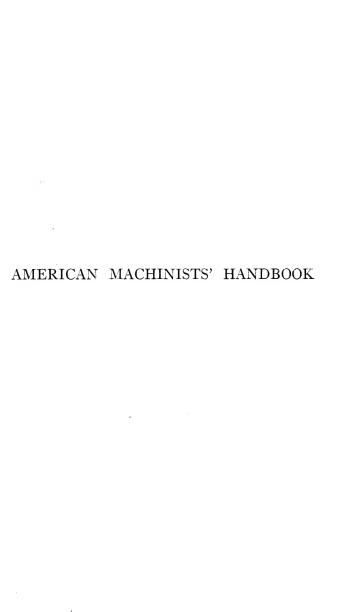
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DICTIONARY OF SHOP TERMS

A REFERENCE BOOK OF MACHINE SHOP AND DRAWING ROOM DATA, METHODS AND DEFINITIONS

BY

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Member American Society of Mechanical Engineers and Franklin Institute
Associate editor of the American Machinist, Author of "Machine
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PREFACE TO SECOND EDITION

EVER since the first printing of this book, the authors have been studying ways and means of making it better, and the addition of one hundred and sixty pages by no means gives an adequate idea of the extent to which it has been revised.

Each section has been carefully studied to make it cover the changed and changing conditions of shop and drawing-room work

in the hope of making it even more valuable than before.

Many of the changes have been due to suggestions made by users of the Handbook, and we shall appreciate a continuance of their interest and assistance in pointing out possibilities of improvement.

The Authors.



PREFACE TO THE FIRST EDITION

EVERY man engaged in mechanical work of any kind, regardless of his position in the shop or drawing room, frequently requires information that is seldom remembered and is not usually available when wanted.

With this in mind it has been our endeavor to present in convenient form such data as will be of value to practical men in the various branches of machine work. While some of the matter included may seem elementary, it was considered necessary in order to make the work complete. Much of the information has never before been available to the mechanic without tiresome search and consultation.

We believe that the Dictionary section will be found of service to the younger mechanics and in helping to establish standard names for various parts which are now more or less confused in different sections of the country.

Our indebtedness to various manufacturers and individuals is hereby acknowledged, and in the back of the book will be found a list of such authorities with page references to the information furnished by them.

We dare not hope that no errors will be found and we shall be glad to have them pointed out and to receive any suggestions as to additions or other changes which may add to the value of the book.

THE AUTHORS.

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THE AMERICAN MACHINISTS' HANDBOOK

SCREW THREADS

CUTTING SCREW THREADS

NEARLY all lathes are geared so that if gears having the same number of teeth are placed on both stud and lead screw, it will cut a thread the same pitch as the lead screw. This is called being geared "even." If the lathe will not do this, then find what thread will be cut with even gears on both stud and lead screw and consider that as the pitch of lead screw. In speaking of the pitch of lead screw it will mean the thread that will be cut with even gears.

In cutting the same thread with even gears, both the work and the lead screw are turning at the same rate. To cut a faster thread, the lead screw must turn faster than the work, so the larger gear goes on the stud and the smaller on the lead screw. To cut a slower thread (finer-pitch or less lead), the larger gear goes on the screw

and the smaller on the stud.

Calling the lead screw 6 to the inch, what gears shall we use to

cut an 8 thread?

Multiply both the lead screw and the thread to be cut by some number (the same number for both) that will give two gears you have in the set. If the gears vary by 4 teeth, try 4 and get 24 and 32 as the gears. If by 5, you get 30 and 40 as the gears. Then as 8 is slower than 6, the large gear goes on the lead screw and the small one on the stud.

Cut an 18 thread with a 5-pitch lead screw and gears varying by 5 teeth. $5 \times 5 = 25$ and $5 \times 18 = 90$. There may not be a 90 gear, but you can use a 2 to 1 compound gear and use a 45 gear instead. That is, put the 25 gear on the stud, use any 2 to 1 com-

bination between this and the 45 gear on the screw.

The 25 gear must drive the large gear of the 2 to 1 combination and the small gear drive the 45-tooth gear, either directly or through

an intermediate.

In cutting fractional threads the same rule holds good. To cut $11\frac{1}{2}$ threads with gears that change by 4 teeth, use $4\times6=24$ and $4\times11\frac{1}{2}=46$, with the 24 gear on the stud and the 46 on the screw. With gears changing by 5 this is not so easy, as $5\times11\frac{1}{2}=57\frac{1}{2}$, an impossible gear. Multiplying by 10 would give 60 and 115, not much better. Multiply by 6 and get $6\times6=36$ and $6\times11\frac{1}{2}=69$, neither of which is in the set. It seems as though 35 and 70 would come pretty near it, but they will cut a 12 thread instead.

To find what thread any two gears will cut, multiply the pitch of lead screw and the gear which goes on it and divide this by the gear on the stud. Suppose we try 40 on the stud and 75 on the lead

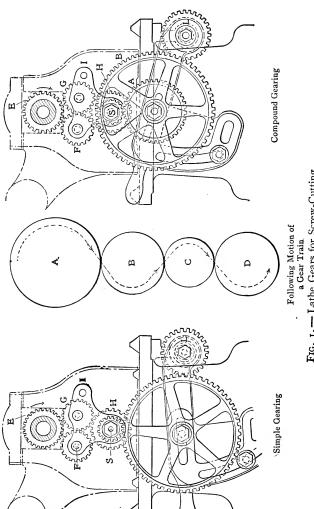


Fig. 1. - Lathe Gears for Screw-Cutting

screw. Multiply 75 by 6=450 and divide by 40 which gives $11\frac{1}{4}$ as the thread that will be cut. Try 45 and 80. $6\times80=480$; divided by $45=10\frac{2}{3}$, showing that the 40 and 75 are nearest and 4 that to cut it exactly a special gear will have to be added to the set. In reality the gears would not change by 5 teeth with a 6-pitch lead screw.

Rules for screw cutting may be summed up as follows, always remembering that the lead screw is the thread that will be cut when gears having the same number of teeth are placed on both screw and stud.

Having	To Find	Rule
A = True lead of screw and $B = $ Thread to be cut	and	Multiply both A and B by any one number that will give gears in the set. Put gear A on stud and gear B on lead screw.
A = True lead of screw $B =$ Thread to be cut $C =$ Gear for stud	D = Gear for screw	Multiply B by \mathcal{L} and divide by A .
A = True lead of screw $B =$ Thread to be cut $D =$ Gear for screw	$C = Gear ext{ for stud}$	Multiply A by D and divide by B
A = True lead of screw $C = $ Gear for stud $D = $ Gear for screw	B = Thread that will be cut	Multiply A by D and divide by C

GEARS FOR SCREW-CUTTING

GEAR trains for screw-cutting are usually arranged similarly to the illustration, Fig. 1. If the gear E on the lathe spindle has the same number of teeth as the gear H on the stud S, the lathe is geared even, i.e., gears having the same teeth placed on both stud and lead screw will cut a thread like the lead screw. As shown, the gears are out of mesh because the tumbler gears F and G do not mesh with E; but moving the handle I down throws F into mesh with E so the drive is through E, F, G, H, E and intermediate to E, driving it so as to cut a right-hand screw if it is a right-hand thread, as is usually the case. Raising handle E cuts out E entirely and reverses the direction of the lead screw.

To follow the motion of a train of gears, take a stick (or your finger if they are not running) and trace the motion from the driver to

the end as shown by the dotted lines in A, B, C and D.

When a lathe is compound geared the stud gear drives an auxiliary gear as A, which multiplies or reduces the motion as the case may be. It will readily be seen, if the stud drives A and B drives L, the motion will be exactly doubled because A has one-half the number of teeth in B.

A SCREW-THREAD ANGLE TABLE

THE accompanying table gives the angle of helix of various pitches and diameters with respect to a line perpendicular to the axis. These angles were worked out with the idea of using them for grinding thread tools for threads of various pitches upon different diameters of work. This table will enable one to set the protractor at the proper angle of side clearance for the work in hand and grind the thread tool correctly without guesswork. This is based on the outside diameter. For coarse and multiple threads it is better to figure on the pitch diameter.





THREAD ANGLE TABLE

THREADS PER INCH = P

Diam. of Work	ı	2	3	4	5	6	7	8	9	10	11	12
1 1 1 1 1 1 1 1 2 2 2 2 3	50°-54 40°-23 32°-30 22°-2 23°-5 20°-4 17°-39 14°-10 13°-4 11°-50 11°-50 10°-26 9°-39 9°-4 8°-8 7°-15 6°-37 6°-4	32°-31 23°-1 117°-41 14°-18 12°-1 10°-20 9°-2 8°-4 7°-12 6°-37 6°-4 6°-36 5°-16 4°-52 4°-34 4°-9 3°-39 3°-19 3°-3	22°-59 15°-48 11°-58 9°-38 8°-8 7°-1 6°-2 34°-48 4°-25 4°-3 3°-15 3°-33 2°-42 2°-26 2°-13 2°-2	17°-39 12°-16 9°-3 7°-15 6°-4 5°-12 4°-33 4°-4 3°-39 3°-19 3°-3 2°-40 2°-18 2°-2 1°-49 1°-49 1°-31	14°-18 9°-39 7°-16 5°-49 4°-52 4°-10 3°-39 3°-15 2°-55 2°-40 2°-26 2°-15 2°-57 1°-57 1°-57 1°-28 1°-22 1°-13	11°-59 8°-8 6°-37 5°-8 4'-3 3°-28 3°-2 2°-42 2°-26 2°-13 2°-2 1°-52 1°-44 1°-37 1°-21 1°-16 1°-7	5°-4° 4°-1° 3°-52 2°-59 2°-36 2°-19 2°-2 1°-54 1°-36 1°-23 1°-18	9°- 2 6°-37 4°-33 3°-52 3°-37 2°-17 2°-2 1°-50 1°-36 1°-13 1°-18 1°-18 1°-1 55′,	8°- 3 5°-23 4°- 3 3°-15 2°-43 2°-19 2°- 2 1°-44 1°-29 1°-15 1°-10 1°- 5 1°-10 1°- 5 1'-10	7°-58 5°-20 4°-1 3°-13 2°-41 1°-28 1°-28 1°-20 1°-14 1°-8 1°-8 1°-0 53' 48' 40'	7°-11 4°-49 3°-37 2°-54 2°-19 1°-27 1°-27 1°-13 1°-7 1°-2 58' 54' 49' 43' 40'	6°- 3 4°-24 3°- 3 2°-26 2°- 2 1°-50 1°-13 1°- 6 1°- 1 56', 53', 49', 41', 37', 33',

While the table is worked out for single threads, it can be used for double or triple threads by considering the lead equal to the advance of the work in one revolution instead of $\frac{1}{p}$, as given in the table.

It is customary in many shops to have several thread tools in stock to cut these various thread angles, each cutting within a certain range of angles. This table will be useful in determining the best range for each thread tool.

$$P = \text{Pitch} = \text{Threads per inch.} \frac{\mathbf{r}}{P} = \text{Lead} = L$$

D = Diameter of work in inches. $\pi = 3.1416 = \frac{C}{D}$ $C = \text{Circumference of Work in inches} = \pi D$

 $C = Circumference of Work in inches = \pi D$

$$\frac{L}{C} = \frac{\text{Lead}}{\text{Circumference of Work}} = \frac{\frac{1}{P}}{\pi D} = \text{Tangent of Angle}$$

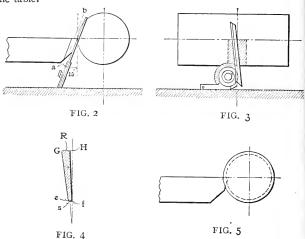
Find Angle in Table of Tangents

THREAD ANGLE TABLE

THREADS PER INCH = P

Diam. of Work	13	14	15	16	18	20	22	24	26	28	30	32
1 4 dia - [valuation of the last of the l	5°-36 3°-45 2°-49 2°-12 1°-53 1°-37 1°-24 1°-7 1°-7 1°-7 1°-1 56' 48' 45' 37' 34' 31'	5°-12 3°-28 2°-36 2°-5 1°-44 1°-30 1°-18 1°-11 1°-3 57' 52' 48' 45' 42' 40' 35' 31' 28'	4°-51 3°-14 2°-26 1°-57 1°-37 1°-24 1°-13 1°-5 59' 45' 42' 39' 36' 32' 29' 24'	4°-33 3°-3 2°-17 1°-48 1°-31 1°-18 1°-18 1°-1 54' 46' 40' 37' 34' 30' 27', 25'	4°- 3 2°-43 2°- 2 1°-37 1°-10 1°- 1 54', 44', 40', 37', 35', 33', 31', 22', 20'	3°-39 2°-26 1°-49 1°-28 1°-13 1°-3 55' 49' 44' 36' 36' 36' 31', 22' 22' 18'	3°-19 2°-13 1°-40 1°-20 1°-7 57' 50' 40' 36' 33' 28' 27' 22' 20' 18'	3°- 1 2°- 1 1°-31 1°- 1 53' 45' 40' 37' 38' 28' 26' 22' 20' 18' 17'	2°-45 1°-53 1°-24 1°-8 57 49 42 38 34 31 28 26 24 21 19 17 16	2°-36 1°-44 1°-18 1°-3 53', 45', 39', 35', 31', 28', 26', 21', 18', 15', 14',	2° 25 1° -37 1° -13 59 49′ 42′ 36′ 32′ 27′ 24′ 16′ 18′ 116′ 113′ 12′	2°-17 1°-31 1°-8 54'446 33'34'33'227'25'23'110'115'115'115'115'115'115'115'115'115

Figs. 2 and 3 show side and front elevations of the thread tool and of the protractor as applied to obtain the proper angle of side clearance to cut a right-hand screw thread. The front edge of the thread tool is used to determine the angle of side clearance. Fig. 4 shows a section taken along the line a b, Fig. 2. It will be noticed that line e f is shorter than G H to give clearance to the cutting edges of the thread tool, and also that G R is equal to H R and e S is equal to f S. The angle of the helix at half the depth of the thread, Fig. 5, can be used, if desired, and can be approximated to from the table, or figured exactly by the method given at the top of the table.



The Use of the Protractor

METRIC THREADS

Metric threads are measured in millimeter but are calculated by the threads per centimeter. Any lathe with a pair of compound or "translating gears" with 50 and 127 teeth, can cut metric threads, the large gear being driven from the stud. Then the gears for the number of threads per centimeter are figured the same as threads per inch as on page 3.

MULTIPLE THREAD CUTTING

THE accompanying table will be found useful when cutting multiple threads. When one thread is cut, the feed nut may be opened (the spindle of course being stopped) and the carriage moved along by hand the distance given in the table; the nut is then closed on the screw and the next thread cut. This is a quick and sure method of starting the second, third or fourth thread where the lead screw of the lathe is of the pitch given in the table.

TABLE FOR MULTIPLE THREAD CUTTING

Cut	Thread on Lead Screw	Move Carriage
DOUBLE		
I	Even	½ inch
114	Any	2 inch
$1\frac{1}{2}$	Any	ı inch
2	4	½ inch
21	Any	2 inch
2 14 2 3 3 3 4 4 4 4 4 5 5 5 5 5 5	Any	ı inch
2 2	Even	½ inch
31	Any	2 inch
21	Any	ı inch
32	8	½ inch
41	Any	2 inch
11	Any	ı inch
42 E	Even	½ inch
51	Any	ı inch
32	,	
TRIPLE		
I	6	3" or 2 threads on lead screw
11	6	13" or 8 threads on lead screw
1 1 2	6	$\frac{2}{3}$ " or 4 threads on lead screw
2	6	1 or 1 thread on lead screw
$2\frac{1}{4}$	6	13" or 8 threads on lead screw
$2\frac{1}{2}$	6	3" or 4 threads on lead screw
-2		3
QUADRUPLE		
I	4	1 inch
114	Any	τ inch
112	Even	½ inch
2	8	inch
21	Any	ı inch
21	Even	½ inch
~		•

To cut a double thread screw $3\frac{1}{2}$ to the inch: the lathe must be geared the same as for a single, triple or quadruple thread. The tool will of course have to be the same width and the depth of cut the same as for a 7 per inch screw. After the first thread is cut it will appear very shallow and wide. With the lathe spindle idle, the nut is opened and the carriage moved (in either direction) I inch; the nut is then closed on the lead screw and the tool is in the proper position to make the second cut.

If the carriage were moved 2 inches, the tool could follow exactly the first groove cut. In the case of a triple-thread screw, if the carriage were moved 3 inches, the tool would follow its original path, and it would do the same in the case of a quadruple thread if moved

4 inches.

The carriage can, of course, be moved $\mathbf{1}$ inch and the nut closed no matter what the pitch of the lead screw may be (unless it is fractional), but in order to close the nut after moving $\frac{1}{2}$ inch, the screw must have some even number of threads per inch.

As will be seen by referring to the table, a lead screw with any even number of threads per inch is used in a number of cases, while in several other instances the screw may be of any pitch—either odd or even. In certain cases 4 and 8 per inch lead screws are specified; and in cutting triple threads a 6 per inch screw is required.

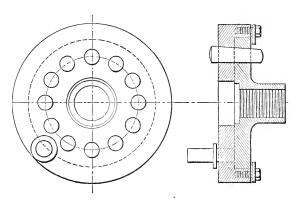


Fig. 6. — Face-Plate for Multiple Thread Cutting

FACE-PLATE FOR MULTIPLE THREAD CUTTING

FIG. 6 shows a face-plate fixture used on various numbers of threads. On an ordinary driving plate is fitted a plate having, as shown, twelve holes enabling one to get two, three, four or six leads if required. This ring carries the driving stud, and is clamped at the back of the plate by two bolts as an extra safeguard. All that is necessary in operation is to slack off the bolts, withdraw the index pin, move the plate the number of holes required, and re-tighten the bolts. It is used on different lathes, as occasion requires, by making the driving plates alike and drilling a hole for the index pin. It is found that the index pin works best when made taper, and a light tap is sufficient to loosen or fix it.

CUTTING DIAMETRAL PITCH WORMS IN THE LATHE

THE accompanying table is to be used in cases where fractional worm-thread cutting is necessary for diametrical pitch worm threads to mesh into diametral-pitch worm gears.

TABLE OF CHANGE GEARS FOR DIAMETRAL PITCH WORMS



_	1	1 +1		l							
Diametral Pitch	हु. प ु	Width of ool Point A	Width of Top of Thread B			Pi	tch of I	Lead Sc	rew		
iametr Pitch	Single Depth	P P	idt. Pop irea		1			1		1	
Ä_	0.7	Wid Tool	≥L!	2	3	4	5	6	7	8	10
2	1.078"	.487"	. 526"	2 2 7	33	4,1	55	66	77.	88	110
$2\frac{1}{2}$.862"	. 390"	.421"	32	132	176	4,4	254 35	44	352	440
3	.719"	. 325"	.350,,	21	22	88 21	110 21	3,4	22	276	220 21
$3\frac{1}{2}$.616"	. 278"	.300#	8 8 4 9	132 49	176 49	220 49	264 49	44	352 49	440
4	.540"	.243"	. 263"	1,1	33 14	2,2	55	33	11	4,4	5,5
5	.431"	.195"	.210"	4 4 3 5	<u> 5 5</u> 3 5	88	22	132	22	176	44
6	. 360″	.162"	.175"	2 2 2 1	4	21	55 21	22	131	88	110
7	. 308″	.139"	. 150"	44	<u>66</u>	88 49	110	132	22	178 49	220 49
8	. 270"	. 122"	.131"	$\frac{1}{1}\frac{1}{4}$	33 28	1,1	55 28	33 14	1,1	2,2 7	55 14
9	. 240"	.108"	.117"	63	22 21	88 63	110	44 21	22	176	22 <u>0</u>
10	.216"	.097"	.105"	220	33 35	44	1,1	6 <u>6</u>	1,1	88	22
11	.196″	.088"	.096"	4	6	8 7	1,0	12	1,4	1,6	2,0
12	.180"	.081"	.088"	$\frac{1}{2}\frac{1}{1}$	11	22 21	55 42	11	1,1	44	55 21
14	. 1 54"	.069"	.075"	22 49	33 49	44	55	65 49	3,1	88 49	110
16	.135"	.061"	.066"	11 28	33 88	22	55	33	7 7 5 6	1,1	55
18	.120"	.054"	.058"	22 63	$\frac{1}{2}\frac{1}{1}$	4.4 63	55 63	22 21	3,1	88	110 63
20	.108"	.048"	.053"	1 1 3 5	33 70	22	11	33	77	44	1,1
24	.090"	.040"	.044"	$\frac{1}{4}\frac{1}{2}$	11	21	55	33	77	22 21	55 42
28	.077"	.034"	.038"	45	33	22 49	55	33	11	44	55 49
32	.067"	.030"	.033"	$\frac{1}{5}\frac{1}{6}$	33 112	11 28	55 112	33	1 1 1 6	114	55
40	.054"	.024"	.026"	70	$\frac{33}{140}$	35	$\frac{11}{28}$	33	1 1 2 0	22	11
48	.045"	.020"	.022"	81	11 56	$\frac{1}{4}\frac{1}{2}$	55 168	33 84	11	$\frac{1}{2}\frac{1}{1}$	55
<u>—</u>											

Formula: ${22 \times \text{Lead Screw} \atop 7 \times \text{Diametral Pitch}} = \text{Ratio of Wheels.}$

355 to 113 is more accurate but requires odd gears; a 71 tooth

gear and a 5 to 1 compound gives 355.

In the first column is found the diametral pitch to be cut. In the second column is found the corresponding single depth of the worm thread. Under the third column is found the width of the tool at the point, the tool being the regular 29-degree included angle. In the fourth column is found the width at the top of the worm thread.

The next heading in the chart is "Pitch of lead screw," and here

are found different pitches of lead screws from 2 to 10.

Example: Suppose it is desired to cut a worm thread of 4 diametral pitch on a single-geared lathe having a 6-pitch lead screw. Now, opposite 4 in the first column find the single depth of worm thread, or 0.540 inch; and continuing in the same direction from left to right, under the next column find the width of the worm-thread tool at the point or end, which is 0.243 inch, and so on to the next column where is found the width of the worm thread at the top, which is 0.263 inch. Say there is a 6-pitch lead screw on the lathe. Then follow right, on in the same direction until coming to the square under 6, and the gear, will be in the ratio of $\frac{3}{7}$. Of course there is no 7 gear on the lathe, so simply bring the fraction 33 to higher denominations, say, $\frac{33}{7} \times \frac{3}{3} = \frac{99}{31}$: that is, put the 99 gear on the spindle or stud, and the 21 gear on the screw. Then use a gear of any convenient size to act as an intermediate gear, and thus connect the gear on the spindle with the gear on the screw. Taking the fraction \(\frac{3}{3} \) and multiplying the numerator and denominator by 4, would give $\frac{13.2}{2.8}$ as the two gears to be used. It will be seen that this last fraction simply changes the number of teeth in the gears, but does not change the value of the fraction; thus there is the same ratio

Take another case: Suppose it is desired to cut a 20-diametral pitch worm thread in a lathe having a 4-pitch lead screw. What would be the necessary gears to cut the desired thread? Next to 20 in the first column is found the single depth of the worm thread, which is 0.108 inch. Continuing on, reading from left to right as in the first case, and 0.048 inch is found as the width of the tool at the point. In the next column is found the width at the top of the worm thread, which in this case is 0.053 inch. Under column 4, and opposite 20, are found the gears necessary for cutting a 20-diametral pitch worm thread in a lathe with a 4-pitch lead screw. The gears

thus found, namely, $\frac{22 \text{ stud}}{35 \text{ screw}}$ may not be in the regular set of gears furnished with the lathe. In that case double up on both and make it $\frac{44 \text{ stud}}{70 \text{ screw}}$, which is the same in value. The two examples thus

70 screw worked out could have been cut on lathes with lead screws having any number of threads per inch, with the same result. One point in cutting these threads is that the tool must be of exact dimensions all over, for if it is not exactly 29 degrees included angle, or the point is not as it should be for width, then there will be an error in the worm thread all around.

THE BROWN & SHARPE 29-DEGREE WORM THREAD AND THE ACME 29-DEGREE STANDARD SCREW THREAD

THERE seems to be some confusion among mechanics regarding the 29-degree Acme standard screw thread and the Brown & Sharpe 29-degree worm thread.

The sketches, Figs. 7 and 8, show plainly the difference between threads of the same pitch in the two systems. The sectional

views are of threads of one-inch linear pitch drawn to scale to the proportions given by the thread formulas in connection with the complete tables of the two systems of threads as given on pages following. The clearance for bottom of thread is the same, o.o.o inch, for all pitches. See formula on page 24 for correct dimensions.

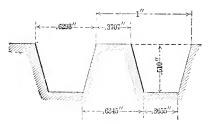


Fig. 7 - Acme 29-Degree Screw Thread

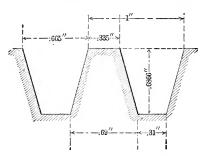


Fig. 8. - Brown & Sharpe 29-Degree Worm Thread

The worm thread is based on the linear pitch of the worm and proportions figured same as rack tooth with varying clearances in bottom. B. & S. 29-deg. screw thread has a uniform clearance of 0.005 inch for all pitches. Do not confuse the two threads.

MEASUREMENT OF V-TOOLS

THE accompanying table of angle measurements should prove of convenience to all who make tools for cutting angles or make the gages for these tools.

The principle here adopted is that, on account of the difficulty and in some cases the impossibility of measuring the tool at its point, the measurement is taken on the angle of the tool at a given distance from the point. In this case the true measurement will be less than the actual measurement by an amount equal to twice the tangent of half of the angle multiplied by the distance of the line of measurement from the point.

For making the measurement the Brown & Sharpe gear-tooth caliper may be used. Fig. 9 shows this tool in position for measuring. The depth vernier A is set to a given depth h, and the measurement is taken by means of the vernier B. The width of the tool point x is equal to the measurement on the line a b less $2h(\tan \frac{C}{a})$. use the table, h is always taken to be $\frac{1}{16}$ inch, which is found to be a convenient depth for most work. If a greater depth is required, all that is necessary is to multiply the figures given by the ratio of the required depth to $\frac{1}{16}$ inch. For instance, if the depth is required

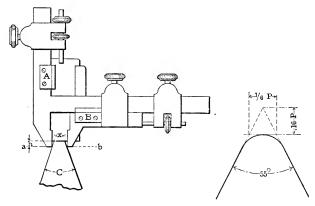


Fig. 9. - Measuring Thread Tools

to be $\frac{1}{8}$ inch, the figures given are multiplied by 2. In the great majority of cases, $\frac{1}{16}$ will be found a suitable value for h, when to find the width of the point x it is merely necessary to deduct the

value of $\frac{tan.\frac{C}{2}}{8}$ for the angle required, which can be obtained at a glance from the table.

In the case of the Sellers or United States standard thread, the point of the tool should be one-eighth of the pitch of the screw, while in the Whitworth standard, as shown, the point of the tool would be one-sixth of the pitch if it were not rounded. By using these figures in combination with the table, it can be determined when sufficient has been ground from the point of the tool.

The table is called "Table for Angle Measurements," because if a sharp angle, that is, one without the point ground away, is measured as above, this measurement, by reference to the table, will give the angle direct.

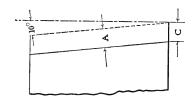
TABLE FOR V-TOOL ANGLE MEASUREMENTS

Degrees	$\frac{tan. \frac{C}{2}}{8}$	Degrees	$\frac{tan.\frac{C}{2}}{8}$	Degrees	$\frac{tan.\frac{C}{2}}{8}$
1 2 3 4 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26 27 28 29	0.0011 0.0022 0.0033 0.0044 0.0055 0.0066 0.0077 0.0088 0.0099 0.0110 0.0121 0.0132 0.0143 0.0154 0.0165 0.0176 0.0187 0.0198 0.0209 0.0220 0.0231 0.0242 0.0253 0.0242 0.0253 0.0264 0.0275 0.0286 0.0298 0.0310 0.0322	31 32 33 34 35 36 37 38 39 40 41 42 43 44 45 46 47 48 49 50 51 52 53 54 55 57 58 57 58	0.0346 0.0358 0.0370 0.0382 0.0394 0.0406 0.0418 0.0430 0.0442 0.0454 0.0466 0.0489 0.0505 0.0518 0.0531 0.0544 0.0557 0.0570 0.0583 0.0596 0.0609 0.0623 0.0637 0.0651 0.0665 0.0679 0.0693 0.0707	61 62 63 64 65 66 67 68 69 71 72 73 74 75 76 77 78 81 82 83 84 85 86 87 88 89	0.0736 0.0751 0.0766 0.0781 0.0796 0.0811 0.0827 0.0843 0.0859 0.0875 0.0998 0.0925 0.0925 0.0976 0.0994 0.1012 0.1030 0.1048 0.1067 0.1086 0.1105 0.1145 0.1165 0.1145
30	0.0334	59 60	0.0721	. 90	0.1250

GRINDING THE FLAT ON THREAD TOOLS

To facilitate grinding the correct width of flat for the single-point inserted tool to cut United States standard form of threads the accompanying table on pages 14 and 15 has been prepared. The distance from the point of the tool to the back is first measured with the micrometer, then the point of the tool may be ground off until the micrometer measurement from the back is equal to the whole depth minus dimension A, when we may be sure, without undertaking the difficult job of measuring it directly, that the flat B has the proper width. The dimensions A and B for pitches from 1 to 64 threads per inch are included in the table.

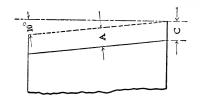
Table for Grinding Flat End of Tool for Cutting U. S. Form of Thread





Threads per Inch	Pitch	A	В	С	Double Depth	Depth
I	1.000	.1064	.125	.1082	1.299	.6495
2	.5000	.0532	.0625	.0541	.6495	• .3247
3	•3333	.0355	.0416	.0360	•433	.2165
4	.2500	.0266	.0312	.0270	.3247	.1623
4 5 6	.2000	.0213	.0250	.0216	.2598	.1299
ð	.1666	.0177	.0208	.0180	.2165	.1082
7 8	.1428	.0152	.0178	.0154	.1855	.0927
8	.1250	.0133	.0156	.0135	.1623	.0812
9	·IIII	.0118	.0138	.0120	.1443	.0721
10	.1000	.0106	.0125	.0108	.1299	.0649
II	.0909	.00963	.0113	.0098	.1180	.0592
12	.0833	.00886	.0104	.0090	.1082	.0541
13	.0769	.00818	.0096	.0083	.0999	.0499
14	.0714	.00758	.0089	.0077	.0920	.0460
15	.0666	.00707	.0083	.0071	.0866	.0433
16	.0625	.00673	.0079	.0068	.0812	.0406
17	.0588	.00620	.0073	.0063	.0764	.0382
18	.0555	-00588	.0069	.0059	.0721	.0360
19	.0526	.00554	.5065	.0056	.0683	.0341
20	.0500	.00530	.0062	.0054	.0649	.0324
21	.0476	.00503	.0059	.0051	.0618	.0309
22	.0454	.0048	.0056	.0049	.0590	.0295
23	.0431	.00451	.0053	.0046	.0564	.0282
24	.0416	.00433	.0052	.0045	.0541	.0270
25	.0400	.00426	.0050	.0043	.0519	.0259
26	.0384	.00409	.0048	.0041	.0491	.0245
27	.0370	.00393	.0046	.0040	.0481	.0240
28	.0357	.00375	.0044	.0038	.0463	.0231
29	.0344	.00366	.0043	.0037	.0447	.0223
30	.0333	.00354	.0041	.0036	.0433	.0216
31	.0322	.00341	.0040	.0035	.0419	.0209
32	.0312	.00332	.0039	.0034	.0405	.0202
	I	I	1	I	1	1

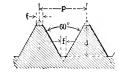
Table for Grinding Flat End of Tool for Cutting U, S. Form of Thread





Threads per Inch	Pitch	A	В	С	Double Depth	Depth
9er Inch 33 34 35 36 37 38 39 40 41 42 43 44 45 46 47 48 49 50 51 52 53 54 55 56 57 58	.0303 .0294 .0285 .0277 .0270 .0263 .0256 .0250 .0243 .0238 .0232 .0227 .0217 .0212 .0208 .0200 .0196 .0198 .0185 .0181 .0178	.00315 .00307 .00295 .00289 .00281 .00272 .00268 .00264 .00255 .00251 .00247 .00238 .00233 .00233 .00225 .00221 .00213 .00208 .00200 .00196 .00192 .00185 .00184	.0037 .0036 .0035 .0034 .0033 .00325 .00303 .00295 .00290 .00283 .00271 .00265 .00265 .00250 .00245 .00240 .00231 .00226 .002218	.0032 .0031 .0030 .00295 .00286 .00282 .00277 .00270 .00262 .00257 .00245 .00245 .00235 .00230 .00212 .00200 .00210 .00200 .00192 .00192 .00189	Depth .0393 .0382 .0370 .0360 .0350 .0341 .0333 .0324 .0319 .0309 .0295 .0295 .0296 .0263 .0258 .0254 .0249 .0245 .02405 .0236 .0228	.0196 .0191 .0185 .0180 .0175 .0170 .0166 .0162 .0159 .01545 .0147 .0147 .0141 .0137 .0131 .0129 .0127 .01245 .01225 .01202 .0118 .0116
59 60	.0169 .0166	.00180	.00211	.00183	.02201	.0110
61	.0163	.00177	.00203	.00130	.02103	.01059
62	.0161	.00173	.00203	.00175	.02095	.01047
63	.0158	.00169	.00198	.00171	.02061	.01030
64	.0156	.00167	.00196	.00169	.02029	.01014

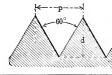
TABLE OF U. S. STANDARD SCREW THREADS



 $\begin{aligned} & \text{Formula} \begin{cases} p = \text{Pitch} = \frac{1}{\text{No. Threads per Inch}} \\ d = \text{Depth} = p \times .64952 \\ f = \text{Flat} = \frac{p}{8} \end{aligned}$

Diam. of Screw	Threads to Inch	Pitch	Depth of Thread	Diam. at Root of Thread	Width of Flat
of Screw 145.6 156.20 15.20 1	to Inch 20 18 16 14 13 12 11 10 9 8 7 7 6 6 5 1 4 1 4 4 4	.0500 .0556 .0625 .0714 .0769 .0833 .0909 .1000 .1111 .1250 .1429 .1667 .1667 .1818 .2000 .2000 .2222 .2222	.0325 .0361 .0405 .0404 .0499 .0541 .0591 .0049 .0721 .0812 .0928 .1082 .1181 .1299 .1444 .1444 .1624 .1656	of Thread .185 .2403 .2936 .3447 .4001 .4542 .5069 .6201 .7307 .8376 .9394 1.0644 1.1585 1.2835 1.3888 1.4902 1.6152 1.7113 1.9613 2.1752 2.4252 2.6288	Flat .0063 .0069 .0078 .0089 .0096 .0104 .0114 .0125 .0139 .0156 .0179 .0208 .0228 .0227 .0250 .0250 .0278 .0278 .0313 .0313
3 3 3 3 4 4 4 4 5 5 5 5 5 6	3 3 3 3 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	.2857 .3977 .3333 .3333 .3478 .3636 .3810 .4000 .4000 .4210 .4210	.1856 .1998 .2165 .2165 .2259 .2362 .2474 .2598 .2598 .2735 .2735	2.8788 3.1003 3.3170 3.5670 3.7982 4.0276 4.2551 4.4804 4.7304 4.9530 5.2030 5.4226	.0357 .0385 .0417 .0417 .0435 .0455 .0476 .0500 .0500 .0526 .0526

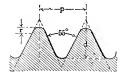
TABLE OF SHARP "V" SCREW THREADS



 $\begin{aligned} & \text{Formula} \begin{cases} p = \text{Pitch} = \frac{1}{\text{No. Threads per Inch}} \\ d = \text{Depth} = p \times .86603 \end{cases} \end{aligned}$

·				
Diam. of Screw	No. Threads per Inch	Pitch	Depth of Thread	Diam. at Root of Thread
of Screw		.0500 .0556 .0625 .0714 .0833 .0833 .0909 .0909 .1000 .1100 .1111 .1111 .1250 .1429 .1429 .14667 .1667 .2000 .2000 .2022 .2222 .2222	Thread .0433 .0481 .0541 .0618 .0722 .0722 .0787 .0866 .0866 .0962 .1083 .1237 .1237 .1443 .1443 .1733 .1733 .1733 .1924 .1924	of Thread .1634 .2163 .2667 .3140 .3557 .4182 .4676 .5301 .5768 .6393 .6826 .7451 .7835 .8776 I.0026 I.0864 I.2114 I.2784 I.4902 I.6152 I.7402
2 2 2 2 2 2 2 3 3 3 3 3 3 3 3 3 3 4	4½ 4 4 4 4 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3	.2222 .2222 .2500 .2500 .2500 .2857 .2857 .2857 .3977 .3977 .3977 .3333 .3333 .3333	.1924 .1924 .2165 .2165 .2165 .2165 .2165 .2474 .2474 .2474 .2666 .2666 .2666 .2886 .2886 .2886	1.8652 1.9902 2.0670 2.1920 2.3170 2.4420 2.5052 2.6301 2.7551 2.8418 2.9668 3.0018 3.1727 3.2077 3.4227

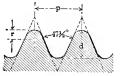
TABLE OF WHITWORTH STANDARD SCREW THREADS



 $\label{eq:posterior} \begin{cases} p = Pitch = \frac{1}{No. \ Threads \ per \ Inch} \\ d = Depth = p \times .64033 \\ r = Radius = p \times .1373 \end{cases}$

Diam. of Screw	No. of Threads per inch	Pitch	Depth of Thread	Diam. at Root of Thread
145 T 98 T 5 25 T 5 24 T 5 7 8 1 14 8 24 12 24 2 24 3 3 3 3 4 4 5 5 5 6	20 18 16 14 12 11 10 10 9 8 7 7 6 6 5 5 4 4 4 4 3 3 4 4 4 3 3 4 4 3 3 4 4 4 4 3 3 4 4 4 4 4 4 4 4 4 4 4 4 4	.0500 .0556 .0625 .0714 .0833 .0833 .0909 .0909 .1000 .1000 .1111 .1250 .1429 .1429 .1429 .1667 .1667 .2000 .2000 .2022 .2500 .2857 .2857 .3077 .3333 .3333 .3478 .3636 .3810 .4900	.0320 .0356 .0400 .0457 .0534 .0534 .0582 .0582 .0640 .0640 .0711 .0800 .0915 .0915 .1067 .1067 .1281 .1281 .1281 .1423 .1601 .1830 .1970 .1970 .2134 .2134 .2227 .2328 .2439 .2501	.1860 .2414 .2950 .3460 .3933 .4558 .5086 .5711 .6219 .6844 .7327 .8399 .9420 1.0670 1.1616 1.2866 1.3689 1.4939 1.7154 1.9298 2.1798 2.3841 2.6341 2.8560 3.1060 3.3231 3.5731 4.0546 4.5343 5.0121 5.4877

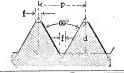
TABLE OF BRITISH ASSOCIATION SCREW THREADS



Formula $\begin{cases} p = \text{Pitch} \\ d = \text{Depth} = p \times . \\ r = \text{Radius} = \frac{2 \times p}{11} \end{cases}$

Num- ber	Diam. of Screw mm.	Approxi- mate Diam. Inches	Pitch mm.	Approximate Pitch Inches	Depth of Thread mm.	Diam. at Root of Thread wm.
0 1 2 3 4 5 6 7 8 9	6.0 5.3 4.7 4.1 3.6 3.2 2.8 2.5 2.2 1.9	.236 .209 .185 .161 .142 .126 .110 .098 .087	1.0 .9 .81 .73 .66 .59 .53 .48 .43 .39 .35	.0394 .0354 .0319 .0287 .0260 .0232 .0209 .0189 .0169 .0154	.6 .54 .485 .44 .395 .355 .32 .29 .26 .235	4.8 4.22 3.73 3.22 2.81 2.49 2.16 1.92 1.68 1.43 1.28
11 12 13 14 15 16 17 18 19 20 21 22 23 24 25	1.5 1.3 1.2 1.0 .9 .79 .70 .62 .54 .48 .42 .37 .33 .29 .25	.059 .051 .047 .039 .035 .031 .028 .024 .021 .019 .017 .015	.31 .28 .25 .23 .21 .19 .17 .15 .14 .12 .11 .10	.0122 .0110 .0098 .0091 .0083 .0075 .0067 .0059 .0055 .0047 .0043 .0039 .0035	.185 .17 .15 .14 .125 .115 .00 .085 .07 .065 .06	1.13 .96 .9 .72 .65 .56 .50 .44 .37 .34 .29 .25 .22

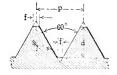
20 FRENCH (METRIC) STANDARD SCREW THREADS



Formula $\begin{cases} p = Pitch \\ d = Depth = p \times .6495 \end{cases}$ $f = Flat = \frac{p}{8}$

Diameter of	Pitch	Diameter at Root	Width of
Screw mm.	mm.	of Thread mm.	Flat mm.
		2 27	.06
3	0.5	2.35	
4	0.75	3.03	.09
5	0.75	4.03	.00
6	1.0	4.70	.13
7	1.0	5.70	.13
8	1.0	6.70	.13
8	1.25	6.38	.16
9	1.0	7.70	.13
	l .		.16
9	1.25	7.38	i
10	1.5	8.05	.19
11	1.5	9.05	.10
12	1.5	10.05	.19
12	1.75	9.73	.22
14	2.0	11.40	.25
16	2.0	13.40	.25
18	2.5	14.75	.31
20	2.5	16.75	.31
22	2.5	18.75	.31
22	3.0	18.10	.38
2.1	3.0	20.10	.38
26		ľ	.38
	3.0	22.10	1 .30
27	3.0	23.10	.38
28	3.0	24.10	.38
30	3.5	25.45	•44
32		27.45	.44
	3.5		1
33	3.5	28.45	•44
34	3.5	29.45	•44
36	4.0	30.80	-5
. 38	4.0	32.80	.5
		32.80	
39	4.0	33.80	•5
40	4.0	34.80	-5
42	4.5	36.15	.56
44	4.5	38.15	.56
		39.15	.56
45	4.5		16.30
46	4.5	40.15	.56
48	5.0	41.51	.63
50	5.0	43.51	.63
52		45.51	.63
-6 ³⁻²	5.0	45.5	.60
56	5.5	48.86	
60	5.5	52.86	.69
64	6.0	56.21	•75
68	6.0	60.21	.75
			.81
72	6.5	63.56	
76	6.5	67.56	.8r
8o `	7.0	70.91	.88
	• •	• •	•

TABLE OF INTERNATIONAL STANDARD SCREW THREADS DIMENSIONS IN MILLIMETERS



Formula
$$\begin{cases} p = \text{Pitch} \\ d = \text{Depth} = p \times .64952 \\ f = \text{Flat} = \frac{p}{8} \end{cases}$$

Diam. of Screw	Pitch	Diam. of Screw	Pitch	Diam. of Screw	Pitch	Diam. of Screw	Pitch
6 7 8 9 10 11 12 14	1.00 1.00 1.25 1.25 1.50 1.50 1.75 2.00 2.00	18 20 22 24 27 30 33 36	2.50 2.50 2.50 3.00 3.00 3.50 4.00	39 42 45 48 52 56 60 64	4.00 4.50 4.50 5.00 5.50 5.50 6.00	68 72 76 80 88 96 116 136	6.00 6.50 6.50 7.00 7.50 8.00 9.00

The "International Standard" is the same, with modifications noted, as that now in general use in France.

INTERNATIONAL STANDARD THREADS

At the "Congress International pour L'Unification des Filetages," held in Zurich, October 24, 1898, the following resolutions were adopted:

The Congress has undertaken the task of unifying the threads of machine screws. It recommends to all those who wish to adopt the metric system of threads to make use of the proposed system. This system is the one which has been established by the "Society for the Encouragement of National Industries," with the following modification adopted by this Congress.

1. The clearance at the bottom of thread shall not exceed $\frac{1}{16}$ part of the hight of the original triangle. The shape of the bottom of the thread resulting from said clearance is left to the judgment of the manufacturers. However, the Congress recommends rounded

profile for said bottom.

3. The table for Standard Diameters accepted is the one which has been proposed by the Swiss Committee of Action. (This table is given above.) It is to be noticed especially that 1.25 mm. pitch is adopted for 8 mm. diameter, and 1.75 mm. pitch for 12 mm. diameter. The pitches of sizes between standard diameters indicated in the table are to be the same as for the next smaller standard diameter.

TABLE OF GERMAN LÖWENHERZ THREAD COMMONLY USED ON THE

The table gives sizes for the German Löwenherz thread, which is used very extensively for fine threads on measuring instruments and similar work in Germany. It was designed by Doctor Löwenherz, Berlin, in 1804. CONTINENT IN FINE WORK

í	s ·	Approx. Inch Size	025	033	038	047	0.058	090	077	100	tor	811	134	152	991	181	215	248	280	315	382	449	919	201	929	902	783	840	616	010
	Tap Drills	Ap	ö	ö —	o	ó	ó	o	o	ö	o -	o	ó	ó	o	o	ó	ó	ó	oʻ	ó	ó	oʻ	o	ō	ó	0	ö	ō	ō
	Taj	Mm.	0.64	0.84	0.07	1.20	1.44	1.74	1.97	2.30	2.65	3.00	3.43	3.87	4.23	4.60	5-45	6.30	7.15	8.00	9.70	11.40	13.10	14.82	16.52	17.93	19.93	21.35	23.35	24.75
	Width of Flat	Inches(F)	0,0012	0.0012	0.0015	0.0017	0.003	0.003	0.0022	0.0025	0.0029	0.0034	0.0037	0.0039	0.0044	0.0049	0.0054	0.0059	0.0064	6900.0	0.0079	0.0089	0.0098	0.0108	0.0118	0.0138	0.0138	0.0157	0.0157	0.0177
,	Pitch Diameter	Inches	0.0320	0.0398	0.0463	0.0566	0.0669	0.07874	0.0800	0.1031	0.1201	0.1367	0.1547	0.1732	0.1000	0.2067	0.2431	0.2795	0.3159	0.3525	0.4252	0.4984	0.5700	0.6442	0.7165	0.7835	0.8622	0.9291	1.0079	1.0748
	Pitch I	Mm.	0.812	1.012	1.175	1.437	1.700	2.000	2.260	2.620	3.050	3.470	3.930	4.400	4.825	5.250	6.175	7.100	8.025	8.954	10.800	12.650	14.500	16.350	18.200	IO.000	21.900	23.600	25.600	27.300
	iameter	Inches	0.0246	0.0325	0.0374	0.0463	0.0551	6990.0	0.0758	0.0886	0.1024	0.1161	0.1329	0.1496	0.1634	0.1772	0.2106	0.2441	0.2776	0.3110	0.3780	0.4449	0.5118	0.5787	0.6457	0.7008	0.7795	0.8346	0.9134	0.9685
	Root Diameter	Mm.	0.625	0.825	0.950	1.175	1.400	1.700	1.925	2.250	2.600	2.950	3.375	3.800	4.150	4.500	5.350	6.200	7.050	7.900	0.600	11.300	13.000	14.700	16.400	17.800	19.800	21.200	23.200	24.600
	Approx. No. of	I nds. per Inch	9.101	9.101	84.7	72.6	63.5	63.5	56.4	50.8	42.3	36.3	33.9	31.7	28.2	25.4	23.1	21.1	19.5	18.1	15.9	14.1	12.7	11.5	10.6	1.6	1.6	7.9	7.9	7.1
	Pitch	Inches	8600.0	0.0098	8110.0	0.0138	0.0157	0.0157	0.0177	0.0197	0.0236	0.0276	0.0295	0.0315	0.0354	0.0394	0.0433	0.0472	0.0512	0.0551	0.0630	0.0700	0.0787	9980.0	0.0945	0.1102	0.1102	0.126	0.126	0.1417
, ,,,		Mm.	0.25	0.25	0.30	0.35	0.40	0.40	0.45	0.30	0.00	0.70	0.75	0.80	0.00	8.1	01.1	1.20	1.30	1.40	1.60	1.80	2.00	2.20	2.40	2.80	2.80	3.20	3.20	3.60
cocci romone, romi, in roge	Diameter	Inches	0.03937	0.04724	0.05512	0.00693	0.07874	0.09055	0.10236	0.11811	0.13780	0.15748	0.17717	0.19685	0.21054	0.23622	0.27500	0.31496	0.35433	0.39370	0.47244	0.55118	0.02002	0.70866	0.78740	0.86614	0.94498	1.02362	1.10236	01181.1
	Dia	Mm.	0.1	1.2	4.1	1.7	5.0	2.3	2.0	3.0	3.5	0.	4.5	'n	5.5	0.0	0.7	6.0	0.6	o i	12	14	oï Oï	81	20	22	24	50	28	30

ROLLED THREADS

The rolled thread process dates back more than 50 years and was first patented in England. It was first used on comparatively rough work such as track bolts but has come to be used on such fine work as the sizing of taps and screws for micrometers.

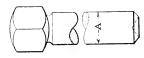
The thread is forced up into the dies so that the finished screw is larger than the original wire by about the depth of one thread. In this way the size of the wire to use for any screw may be found by subtracting the depth of one thread from the outside diameter of

the screw. This is $\frac{.866}{\text{threads per inch}}$. Exact allowance depends on

material being rolled and other conditions.

The dies are usually flat plates of steel, having grooves of the same pitch and shape as the thread to be rolled. The dies can be easily laid out as follows:

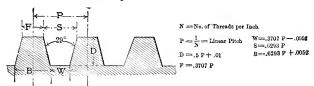
Draw a horizontal line equal in length to the circumference of the wire or blank, and at its end draw a vertical line equal to the lead of the screw. The diagonal line made by joining these two points shows the angle of incline of the grooves. This can be done more easily if both the circumference and the pitch are laid out to ten times their actual dimensions.



DIMENSIONS OF BLANKS FOR U. S. S. ROLLED THREAD SCREWS (REED & PRINCE MFG. Co.)

Size	T. P. I.	A	Size	T. P. I.	A
18 8 16 14 4 5 6 7 16 12 2	40 24 20 18 16 14	.1074 .1054 .1586 .1586 .2157 .2137 2745 .2715 .3325 .3295 .3890 .3860 .4480	$\frac{9}{16}$ $\frac{5}{8}$ $\frac{3}{4}$ $\frac{7}{8}$ $\frac{1}{4}$ $\frac{1}{4}$	12 11 10 9 8 7	.5063 .5031 .5038 .5065 .6828 .6794 .8006 .7972 .9165 .9131 I.0298 I.0262 I.1548 I.1512

ACME 20° SCREW THREADS



The Acme standard thread is an adaptation of the most commonly used style of Worm Thread and is intended to take the place of the square thread.

It is a little shallower than the worm thread, but the same depth

as the square thread and much stronger than the latter.

The various parts of the Acme standard thread are obtained as follows:

Width of Point of Tool for Screw Thread =

$$\frac{.3707}{\text{No. of Threads per inch}}$$
 - .0052.

Width of Screw or Nut Thread = $\frac{3707}{\text{No. of Threads per inch}}$.

Diameter of Screw at Root =

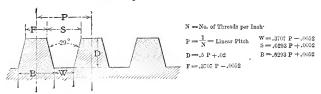
Diameter of Screw
$$-\left(\frac{1}{\text{No. of Threads per inch}} + .020\right)$$
.

Depth of Thread = $\frac{1}{2 \times \text{No. of Threads per inch}} + .oro.$

Table of Acme 29° Screw Thread Parts

N	P	D	F	w	s	В
Number of Threads per Inch	Pitch of Single Thread	Depth of Thread	Width of Top of Thread	Width of Space at Bottom of Thread	Width of Space at Top of Thread	Thickness at Root of Thread
I	1.0	.5100	.3707	.3655	.6293	.6345
$1\frac{1}{3}$.750	.3850	.2780	.2728	.4720	.4772
2	.500	.2600	.1853	.1801	.3147	.3199
3	-3333	.1767	.1235	.1183	.2098	.2150
4	.250	.1350	.0927	.0875	.1573	.1625
5 6	.200	.1100	.0741	.0689	.1259	.1311
6	.1666	.0933	.0618	.0566	.1049	.1101
7 8	.1428	.0814	.0529	.0478	.0899	.0951
8	.125	.0725	.0463	.0411	.0787	.0839
9	.1111	.0655	.0413	.0361	.0699	.0751
10	.10	.0600	.0371	.0319	.0629	.0681

ACME 20° TAP THREADS



THE Acme standard tap-thread is cut with the same width of tool as the screw-thread and the diameter at the root is the same for tap and screw. Clearance at bottom of thread between screw and nut is obtained by boring the nut blank .020 oversize.

The outside diameter of the tap is made .020 larger than the screw to give clearance between top of screw-thread and bottom of nut.

Width of Thread =
$$\frac{.3707}{\text{No. of Threads per Inch}}$$
 - .0052

Diameter of Tap = Diameter of Screw + .020.

Diameter of Tap at Root =
$$\frac{I}{\text{No. of Threads per Inch}} + .040.$$

Depth of Thread =
$$\frac{1}{2 \times \text{No. of Threads per Inch}} + .020$$
.

TABLE OF ACME STANDARD 29° TAP-THREAD PARTS

N.	P	D	F	w	S	В
Number of Threads per Inch	Pitch of Single Thread	Depth of Thread	Width of Top of Thread	Width of Space at Bottom of Thread	Width of Space at Top of Thread	Thickness at Root of Thread
1 1 1 1 3 2 3 4 5 6 7	1.0 .750 .500 .3333 .250 .200 .1666 .1428	.5200 .3950 .2700 .1867 .1450 .1200 .1033 .0914	.3655 .2728 .1801 .1183 .0875 .0689 .0566 .0478	.3655 .2728 .1801 .1183 .0875 .0689 .0566 .0478	.6345 .4772 .3199 .2150 .1625 .1311 .1101 .0951	.6345 .4772 .3199 .2150 .1625 .1311 .1101 .0951
9	.1111	.0755	.0361	.0361	.0751 .c681	.0751 .0681

Brown & Sharpe Screw Thread Micrometer Caliper Readings

READING OF CALIPER

For U. S. Threads =
$$D - \frac{.6495}{P}$$

U. S. STANDARD THREADS

Diam.	Pitch	Caliper Reading		Diam.	Pitch	Caliper Reading	
D	P	$D - \frac{.6495}{P}$.64 <u>95</u> P	D	P	$D - \frac{.6495}{P}$.649 5 P
	64 62 60 58 56 54 52 50 48 44 42 40 38 36 34 32 32 28 26 24 22		.0101 .0105 .0108 .0112 .0116 .0120 .0125 .0135 .0141 .0148 .0155 .0162 .0171 .0180 .0191 .0203 .0217 .0232 .0250	145 P 250 P	20 18 16 14 13 12 11 10 98 7 7 66 5 ^{1/2} 4 ^{1/2} 4 ^{1/2} 3 ^{1/4} 3	.2176 .2765 .3344 .3911 .4501 .5084 .566 .0851 .8029 .9188 I.0322 I.1572 I.2668 I.3918 I.507 I.6201 I.7451 I.8557 2.3376 2.8145 3.3002 3.7835	.0324 .0360 .0406 .0499 .0541 .0590 .0649 .0721 .0812 .0928 .1082 .1180 .1299 .1299 .1299 .1443 .1624 .1855 .1998

As there is no standard of diameter for the finer pitches, the columns for diameter and caliper reading are left blank. The column on the right gives the number to be subtracted from the diameter to obtain the caliper reading.

For explanation of screw thread micrometer caliper, refer to page 28.

Brown & Sharpe Screw Thread Micrometer Caliper Readings

READING OF CALIPER

For "V" Threads =
$$D - \frac{.866}{P}$$

"V" THREADS

Diam.	Pitch	Caliper Reading		Diam.	Pitch	Caliper Reading	
D	P	$D - \frac{.866}{P}$.866 P	D	P	$D - \frac{.866}{P}$.866 P
	64 62 60 58 54 52 50 44 42 40 38 36 34 32 30 28 26	•	.0135 .0140 .0144 .0149 .0155 .0160 .0167 .0180 .0188 .0197 .0206 .0217 .0228 .0241 .0255 .0271 .0289 .0309	14-14-15 - 15 concess. 1-7 17-12-12-12-15 15 15 15 15 15 15 15 15 14 15 15 14 15 15 15 15 15 15 15 15 15 15 15 15 15	24 20 20 18 18 16 16 14 14 13 12 14 12 11 10 10 10 7 6	.2139 .2067 .2069 .2644 .3269 .3209 .3834 .3756 .4334 .4278 .5006 .4903 .5463 .5384 .6009 .6634 .7788 .8918 I.0168 I.1263 I.3557	.0361 .0433 .0481 .0541 .0541 .0541 .0619 .0606 .0722 .0619 .0722 .0787 .0866 .0866 .0866 .0962 .1082 .1082 .1237 .1443

As there is no standard of diameter for the finer pitches, the columns for diameter and caliper reading are left blank. The column on the right gives the number to be subtracted from the diameter to obtain the caliper reading.

For explanation of screw thread micrometer caliper, refer to page 28.

Brown & Sharpe Screw Thread Micrometer Caliper Readings

READING OF CALIPER

For Whitworth Threads = $D - \frac{.640}{P}$

WHITWORTH STANDARD THREADS

Diam.	Pitch	Caliper Reading	
D	P	$D - \frac{.640}{P}$.640 P
14-515-360-15-48-515-36-1-55-515-1-55-515-1-55-515-1-55-515-1-55-515-1-55-515-1-55-515-1-55-515-1-55-515-5	20 18 16 14 12 11 10 10 9 9 8 7 7 6 6 5 4 ¹ / ₂ 4 ¹ / ₂ 4 ¹ / ₂	.2180 .2769 .3350 .3918 .4467 .5092 .5668 .6293 .6860 .7485 .8039 .8664 .9200 1.0336 1.1586 1.2684 1.3934 1.4970 1.6220 1.7328 1.8578 1.9828	.0320 .0355 .0400 .0457 .0533 .0533 .0582 .0582 .0640 .0640 .0711 .0800 .0914 .1066 .1066 .1280 .1280 .1422 .1442
-8	72	2.9020	

SCREW-THREAD MICROMETER CALIPER

THE Brown & Sharpe thread micrometer is fitted with pointed spindle and "V" anvil as in Fig. 10, to measure the actual thread on the cut surface. Enough of the point is removed and the bottom of the "V" is carried low enough so that the anvil and spindle clear the top and bottom of the thread and rest directly on the sides of the thread.

As it measures one-half of the depth of the thread from the top, on each side, the diameter of the thread as indicated by the caliper, or the pitch diameter, is the full size of the thread less the depth of one thread.

This depth may be found as follows;

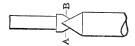


Fig. 10. - Spindle and Anvil of Thread Micrometer

As the U. S. thread is flatted $\frac{1}{8}$ of its own depth on top, it follows that the pitch diameter of the thread is increased $\frac{1}{8}$ on each side, equaling $\frac{1}{4}$ of the whole depth and instead of the constant .866 we use the constant .6495, which is $\frac{3}{4}$ of .866.

When the point and anvil are in contact the o represents a line

drawn through the plane A B, Fig. 10,

and if the caliper is opened, say to .500, it represents the distance of the two planes .500" apart. The preceding tables are used in connection with the micrometer.

MEASURING EXTERNAL SCREW-THREAD DIAMETERS WITH MICROMETERS AND WIRES

It is frequently necessary, especially in making a tap or threadplug gage, to measure the thread diameter on the thread angle in addition to measuring on top of the thread and at the bottom of the thread groove, and unless calipers made expressly for such work are at hand, the measurement on the thread angle is not made with any degree of accuracy or is omitted entirely. The accompanying sketches, Figs. 11, 12 and 13, formulas, and tables, are worked out for convenience in screw-thread inspection, so that by using ordinary micrometer calipers and wire of the diameter called for in the table the standard threads can be compared with the figures given.

Threads of Special Diameter

For threads of special diameter the values of x, x_1 or x_2 can be readily computed from the formula corresponding to the method of measuring to be used. The method shown in Fig. 11 at x is liable to lead to an error unless care be taken that the diameter on top of the thread is correct, and also that the flatted surface on the top of the threads is concentric with the rest of the thread. The concentricity of the flatted surface can be tested by measuring, as at x, Fig. 11, at several points on a plane through the axis and at right angles to it. The wire used must be round and of uniform diameter.

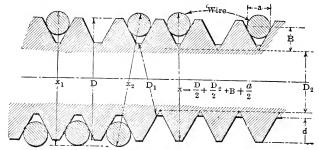


Fig. 11. - Measuring U. S. Standard Threads

D = outside diameter of thread.

 $D_1 = \text{root diameter measured in thread groove.}$

n = number of threads per inch of length.

d = depth of thread.

p = distance from center to center of adjacent threads.

= width of flat on U. S. Standard thread.

top of wire laid in thread groove on opposite side.

a = diameter of wire used.

B = distance from apex of thread angle at root, to center of wire.

 D_2 = diameter of cylinder touched by apexes of thread angles. x = diameter from top of threads on one side of tap or bolt, to

U. S. STANDARD THREAD

$$p = \text{lead} = \frac{1}{n}, \text{ for single threads.}$$

$$d = p \times .6495 = \frac{.6495}{n}.$$

$$D_1 = \sqrt{(D - 2d)^2 + \left(\frac{\text{lead}}{2}\right)^2}.$$

$$f = \frac{p}{8}.$$

$$a = \text{from } p, \text{ max; to } p \times .505, \text{ min.}$$

$$B = \frac{a}{2} \div \sin 30^\circ = a.$$

$$D_2 = D - \frac{\text{I.}5155}{n}.$$

$$\alpha = \frac{D}{2} + \frac{D_2}{2} + B + \frac{a}{2}.$$

$$\alpha_1 = D_2 + 2B + a.$$

$$\alpha_2 = \sqrt{(D_2 + 2B)^2 + \left(\frac{\text{lead}}{2}\right)^2} + a.$$

Table for Measuring U. S. Standard Threads with Micrometers and Wires

D	n	Di	D3	a	В	$\left(\frac{\text{lead}}{2}\right)^2$	x	X ₁	x ₂
1" 4 5" 16 3" 7 16 1	20 18 16 14 13	.1867 .2419 .2954 .3465 .4019	.2283 .2803 .3292	.04 .04 .04 .06	.04 .04	.000625 .000771 .000976 .001274	•2721 •3304 •3876 •4433 •5317	.4003	•3495 •4016
9 7 16 7 15 8 116 7 136 7 136 7	12 11 11 10	.4561 .5089 .5712 .6221 .6844		.06 .06 .06	.06 .06	.001735 .002065 .002065 .0025	.5893 .6461 .7086 .7643 .8267		.6177 .6681 .7312 .7801 .8425
77 8 15 16 17 18 18 14	9 9 8 7 7	.7327 .7950 .8399 .9421 1.0668	.7066 .7691 .8105 .9085	0.10	0.10	.003086 .003086 .003906 .005102	1.0033 1.0553 1.1667	1.1105	1.0706 1.1124 1.2107
13/8 112/1 15/8/1 13/4 17/8	6 6 5 ¹ / ₂ 5 5	1.1614 1.2862 1.3917 1.4935 1.6182	1.2474 1.3494 1.4469	0.10 0.10 0.15 0.15	0.10 0.15 0.15	.006944 .006944 .008263 .010	1.5237 1.7122 1.8234	1.4224 1.5474 1.7994 1.8969 2.0219	1.5497 1.8019 1.8997
$2''$ $2\frac{1}{8}''$ $2\frac{1}{4}''$ $2\frac{3}{8}''$ $2\frac{1}{2}''$	$4\frac{1}{2}$ $4\frac{1}{2}$ $4\frac{1}{2}$ 4	1.7149 1.8393 1.9641 2.0540 2.1787	1.7882 1.9132 1.9961	0.15 0.15 0.15 0.15 0.15	0.15 0.15 0.15	.012343 .012343 .012343 .015625 .015625	2.1816 2.3066 2.4105	2.2382 2.3632	2.2411 2.3667 2.4495
2 ³ / ₄ 3 ¹ / ₃ 3 ¹ / ₄ 3 ¹ / ₂ 3 ³ / ₄	$ \begin{array}{c} 4 \\ 3^{\frac{1}{2}} \\ 3^{\frac{1}{2}} \\ 3^{\frac{1}{4}} \\ 3 \end{array} $	2.6326		0.15 0.20 0.20 0.20 0.20	0.20 0.20 0.20	.015625 .020392 .020392 .023654 .02775	3.0835 3.3335	3.4170 3.6337	3.1704 3.4200 3.6368
4 ¹ / ₄ , 4 ¹ / ₄ , 4 ¹ / ₄ , 4 ³ / ₄ , 5"	3 78 28 28 25 8 2 2 2 2 2 2 2 2 2 2 2 2 2 2	3.5708 3.8019 4.0318 4.2592 4.4848	3.7228 3.9489 4.1728	0.20 0.20 0.20 0.20 0.20	0.20	.03024 .03305 .03625	4.0474 4.2864 4.5244 4.7614 4.9970	4.3228 4.5500 4.7728	4.3264 4.5530 4.7767
5½" 5½" 5½" 6"	2½" 2¾" 2¾" 2¾" 2¼" 2¼"	4.7346 4.9574 5.2072 5.4271	4.8619 5.1119	0.20 0.20 0.20 0.20	0.20	.04431	5.2470 5.4810 5.7310 5.9 6 32	5.4619 5.7119	5.4661 5.7160

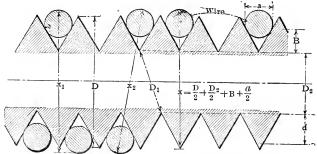


Fig. 12. - Measuring 60-Degree V-Threads

D = outside diameter of thread.

 D_1 = root diameter measured in thread groove.

n = number of threads per inch of length.

d = depth of thread.

p = distance from center to center of adjacent threads.

a = diameter of wire used.

B = distance from apex of thread angle at root, to center of wire.

 D_2 = diameter of cylinder touched by apexes of thread angles.

x = diameter from top of threads on one side of tap or bolt, to top of wire laid in thread groove on opposite side.

$$p = \text{lead} = \frac{1}{n}, \text{ for single threads.}$$

$$d = p \times .866, = \frac{.866}{n}.$$

$$D_1 = \sqrt{(D - 2d)^2 + \left(\frac{\text{lead}}{2}\right)^2}.$$

$$a = \frac{p}{.866}, \text{max; to } p \times .577, \text{ min.}$$

$$B = \frac{a}{2} \div \sin 30^\circ = a.$$

$$D_2 = D - \frac{1.732}{n}.$$

$$x = \frac{D}{2} + \frac{D_2}{2} + B + \frac{a}{2}.$$

$$x_1 = D_2 + 2B + a.$$

$$x_2 = \sqrt{(D_2 + 2B)^2 + \left(\frac{\text{lead}}{2}\right)^2} + a.$$

Table for Measuring 60-Degree V-Threads with Micrometers and Wires

$\begin{array}{cccccccccccccccccccccccccccccccccccc$										
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	x_2	x ₁	x	$\left(\frac{\text{lead}}{2}\right)^2$	В	a	D_2	D ₁	n	D
$ \begin{array}{cccccccccccccccccccccccccccccccccccc$.2846 .3375 .3375 .3375 .5375 .5998 .7115 .8784 .9413 .9843 .0466 .0854 .11800 .1304	.2834 .3363 .3867 .4938 .5357 .5982 .6476 .7100 .8768 .93.3 .9826 1.045 1.3026 1.5361 1.7286 1.8536 1.9400 2.0650 2.1900 2.1900 2.1900 2.1900 2.1900 2.1900 2.1900 2.3150 2.440 2.917 3.105	.2667 .3244 .3808 .4656 .5178 .5803 .6987 .9288 .9912 1.0417 1.1513 1.2763 1.4556 1.5806 1.5806 1.5907 2.0325 2.2825 2.4075 2.5835 2.8335 3.0525 3.3025	.000625 .000771 .0009765 .0001274 .001735 .001735 .0020657 .0025 .003086 .003086 .003102 .005102 .005102 .006944 .010 .010 .012343 .012343 .012343 .012343 .012343 .012343 .012343	0.04 0.04 0.06 0.06 0.06 0.06 0.10 0.10 0.10 0.10	0.04 0.04 0.06 0.06 0.06 0.06 0.10 0.10 0.10 0.15 0.15 0.15 0.15 0.15	.1634 .2163 .2667 .3138 .3557 .4182 .4676 .530 .5768 .6393 .6826 .7450 .7835 .8776 I.0026 I.0863 I.2113 I.2786 I.4036 I.4	.165,3 .2180 .26858 .31580 .4202 .4697 .53199 .57899 .6412 .6847 .7879 .8803 I.0050 I.0895 I.2141 I.4071 I.4941 I.4941 I.4948 I.7435 I.8683 I.9930 2.0707 2.3203 2.5089 2.57587	20 18 16 14 12 11 10 10 9 9 7 7 6 6 5 5 1 ¹ / ₂ 1 ¹ / ₂	14 128 7 1 23 1 1 30 1 1 24 1 1 1 1 1 1 1 1 1 1 1 1 1 1 2 2 2 2

WATCH SCREW THREADS

WATCH screw threads are of sharp V-form and generally 45-degree angle for screws used in nickel and brass; though 60 degrees for use in steel. The Waltham Watch Company and others use the centimeter as the unit for all measurements with the exception of the pitch, which is based on the inch; the Waltham threads being 110, 120, 140, 160, 170, 180, 200, 220, 240, 254, per inch and the diameters ranging from 0.120 to 0.035 cm.

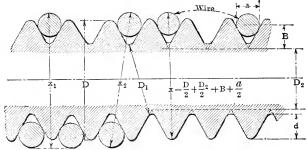


Fig. 13. - Measuring Whitworth Threads

D =outside diameter of thread.

 D_1 = root diameter measured in thread groove.

n = number of threads per inch of length.

d = depth of thread.

p = distance from center to center of adjacent threads.

r = radius on Whitworth thread.

a = diameter of wire used.

B =distance from apex of thread angle at root, to center of wire.

D₂= diameter of cylinder touched by apexes of thread angles.

x = diameter from top of threads on one side of tap or bolt, to top of wire laid in thread groove on opposite side.

WHITWORTH THREAD

$$p = \text{lead} = \frac{1}{n}, \text{ for single threads.}$$

$$d = p \times .64033 = \frac{.64033}{n}.$$

$$D_1 = \sqrt{(D - 2d)^2 + \left(\frac{\text{lead}}{2}\right)^2}.$$

$$r = p \times .1373.$$

$$a = p \times .84, \text{ max; to } p \times .454, \text{ min.}$$

$$B = \frac{a}{2} \div \sin 27^\circ 30' = \frac{a}{.9235}.$$

$$D_2 = D - \frac{1.600825}{n}.$$

$$x = \frac{D}{2} + \frac{D_2}{2} + B + \frac{a}{2}.$$

$$x_1 = D_2 + 2B + a.$$

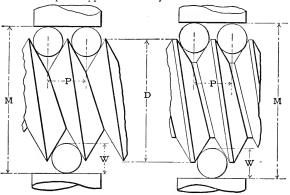
$$x_2 = \sqrt{(D_2 + 2B)^2 + \left(\frac{\text{lead}}{2}\right)^2} + a.$$

Table for Measuring Whitworth Threads with Micrometers and Wires

D	n	D ₁	D_2	a	В	$\left(\frac{\text{lead}}{2}\right)^2$	x	x ₁	x ₂
1" 4 5" 15 3" 8 7 1 6 1"	20 18 16 14 12	.1875 .2428 .2965 .344 .3953	.1699 .2235 .2749 .3231 .3666	0.04	.04331	.000625 .000771 .000976 .001274	•3313	.2965 .3501 .4015 .4497 .5563	.2977 .3514 .4029 .4512 .5582
9 // 16 5 // 16 3 // 1 3 // 1 3 // 1 3 //	12 11 11 10 10	.4576 .5105 .5728 .6239 .6862	.4291 .4794 .5420 .5899 .6524		.06496 .06496 .06496 .06496	.001735 .002065 .002065 .0025	.5907 .6372 .7097 .7649 .8274	.6190 .6693 .7319 .7798 .8422	.6204 .6710 .7334 .7815 .8438
1 6 7 7 8 1 5 7 1 1 6 1 7 1 1 4 7 1 1 4 7 1 1 4 7 1 1 4 7 1 1 1 4 7 1 1 1 1	9 9 8 7 7	.7348 .797 .8422 .9447 1.0693	.6971 •7596 •7999 .8963 1.0213		.06496 .06496 .10839 .10839	.003086 .003086 .003906 .005102	•9435 1.0583 1.169	.8870 .9495 1.1167 1.2131 1.3381	.9512 1.1185 1.2153
$1\frac{3}{8}''$ $1\frac{1}{2}''$ $1\frac{5}{8}''$ $1\frac{3}{4}''$ $1\frac{7}{8}''$	6 6 5 5 4 ¹ / ₂	1.2892 1.3726 1.497	1.1082 1.2332 1.3048 1.4298 1.5193	0.10 0.10 0.15 0.15	.10839 .10839 .16242 .16242	.006944 .006944 .010 .010	1.525 1.7023 1.8273	1.5500 1.7796 1.9046	1.4276 1.5523 1.7826 1.9074 1.9973
$2''$ $2\frac{1}{8}''$ $2\frac{1}{4}''$ $2\frac{3}{8}''$ $2\frac{1}{2}''$	$ \begin{array}{c c} 4\frac{1}{2} \\ 4\frac{1}{2} \\ 4 \\ 4 \\ 4 \\ 4 \end{array} $	1.8437 1.9338	1.6443 1.7693 1.8498 1.9750 2.100	0.15 0.15 0.15	.16242 .16242 .16242 .16242	.012343 .012343 .015625 .015625	2.1845 2.2873 2.4123	2.2441 2.3246 2.4498	2.328
$2\frac{3}{4}''$ $3''$ $3\frac{1}{4}''$ $3\frac{1}{2}''$ $3\frac{3}{4}''$	3½ 3½ 3½ 3¼ 3¼ 3	2.6397 2.860	2.2926 2.5426 2.7574 3.0074 3.2164		.21567 .21567 .21567	.020392 .020392 .023654 .023654	3.087 3.3194 3.5694	3.1740 3.3887 3.6387	2.9276 3.1773 3.3924 3.642 3.8515
$4''$ $4\frac{1}{4}''$ $4\frac{1}{2}''$ $4\frac{3}{4}''$ $5''$	$ \begin{array}{c} 3 \\ 2\frac{7}{8} \\ 2\frac{7}{8} \\ 2\frac{3}{4} \\ 2\frac{3}{4} \end{array} $	3.5768 3.808 4.0582 4.2878 4.5376	4.168	0.20 0.20 0.20	.21567 .21567 .21567 .21567 .21567	.027755 .030241 .030241 .033051 .033051	4.287 4.537 4.7746	4.3243 4.5743	4.578 4.8025
5½" 5½" 5¾" 6"	$ \begin{array}{c} 2\frac{5}{8} \\ 2\frac{5}{8} \\ 2\frac{1}{2} \\ 2\frac{1}{2} \end{array} $	4.7658 5.0156 5.2415 5.4913	4.890 5.110	0.20	.21567 .21567 .21567 .21567	.036252 .036252 .040 .040	5.5107 5.7455	5.2713 5.5213 5.7413 5.9913	5.5248 5.7446

MEASURING FINE PITCH SCREW-THREAD DIAMETERS

THE accompanying table should be of service to those using the three-wire system of measurement as the constants cover the finer pitches and may be easily applied to screw threads of any diameter. The diagrams, Fig. 14, make the method of application so plain that no description appears necessary.



Formulas:

For V Thread D = (M - 3 W) + 1.732 P.M = (D - 1.732 P) + 3 W. For Sellers Thread D = (M - 3 W) + 1.5155 P.M = (D - 1.5155 P) + 3 W.

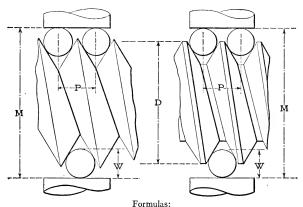
Fig. 14. - Measuring Fine Pitch Threads

Constants for Use with the 3-Wire System of Measuring Screw Threads

Threads per Inch	For V Thread 1.732 P =	For Sellers Thread 1.5155 P=	Threads per Inch	For V Thread 1.732 P =	For Sellers Thread 1.5155 P =
8 9	.21650	.18943	25 28	.06928 .06185	.06062 .05412
10	.17320	.15155	32 36	.05412	.04736
12	•14433 •13323	.12629	40 48	.04330	.03789
14 16	.12371	.10825	50 56	.03464	.03031
18 20	.09622 .08660	.08419 .07578	64 80	.02706 .02165	.02368 .01894
22 24	.07872 .07216	.06889 .06314	100	.01732	.01516

MEASURING METRIC SCREW THREAD DIAMETERS

The tables and formulas given herewith in connection with Fig. 15 should be of value to those engaged in work requiring the frequent production of thread gages or special taps in the metric sizes. The three wire system is used as in the preceding tables, the wires being applied as indicated.



For V Threads

For Threads with Flat Top and Bottom equal to \(\frac{1}{3} \) of

the Pitch

$$D = (M - 3 W) + 1.732 P.$$

 $M = (D - 1.732 P.) + 3 W.$

$$D = (M - 3 W) + 1.5155 P.$$

 $M = (D - 1.5155 P.) + 3 W.$

Fig. 15. — Measuring Metric Threads

CONSTANTS FOR USE IN MEASURING METRIC SCREW THREADS

BY THE 3-WIRE SYSTEM

Pitch m-m	Pitch Inches	1.732P	1.5155P	Pitch m-m	Pitch Inches	1.732P	1.5155P
0.5 0.75 1.0 1.25 1.5 1.75	.01969 .02953 .03937 .04921 .05900 .06890	.03410 .05109 .06819 .08523 .09719	.02984 .04475 .05966 .07458 .08941	4.5 5.0 5.5 6.0 6.5	.17717 .19685 .21654 .23622 .25591	.30686 .34094 .37500 .40913 .43773 .47677	.26850 .29833 .32816 .35799 .38783
2.0 2.5 3.0 3.5 4.0	.07874 .09843 .11811 .13780 .15748	.13638 .16948 .20456 .23867	.11933 .14917 .17899 .20784 .23866	7.5 8.0 9.0	.29528 .31496 .35433 .39370	.51092 .54551 .60870 .68189	.44749 .47732 .53699 .59665

MEASURING ACME 29-DEGREE THREADS

THE diameter of a wire which will be flush with tops of thread on tap when laid in the Acme thread groove, Fig. 16, will be found as follows:

Rad. of wire section = side opp. = side adj. \times tan. 37° 45′ = $\frac{p \times .6293 + .0052}{2} \times .77428$.

Diam. of wire = $(p \times .6293 + .0052)$.77428.

Wires of the diameter given in the table come flush with the tops of tan threads and project .o.o above the top of screw threads.

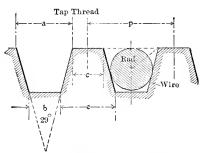


Fig. 16. - Measuring Acme Threads

Table of Wire Sizes for Measuring Acme Standard 29° Screw and Tap Threads

$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	Threads per Inch	Pitch	Diam. of Wire
	1½ 1¾ 2 2½ 3 4 5 6 7 8	.750 .6666 .5774 .500 .400 .3333 .250 .200 .1666 .1428 .125	0.3694 0.3288 0.2824 0.2476 0.1989 0.1664 0.1258 0.1014 0.0852 0.0736 0.0649 0.0581

MEASURING BROWN & SHARPE 29-DEGREE WORM THREADS

The diameter of wire for Brown & Sharpe worm thread, Fig. 17, for each pitch, that will rest in the thread groove on the thread angle and be flush with the tops of the finished threads, is found as follows; Rad. of wire section (see table) = side opp. = side adj. × tan.

 $37^{\circ}46' = \frac{0.665 P}{2} \times 0.77428 = 0.257448 P$ and diam. of wire = 0.5149 P.

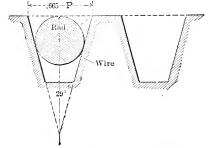


Fig. 17. - Measuring Brown & Sharpe Worm Threads.

Table of Wire Sizes for Measuring B. & S. 29° Worm Threads

Threads per Inch	Pitch	Diam. of Wire			
$\frac{1}{2}$	2,	1.0298			
4 7	1.750	0.9010			
Mir-2473 44h5	1.500	0.7723			
<u>4</u> 5	1.250	0. 6436			
I	1.0	0.5149			
$1\frac{1}{2}$. 666 6	0.3432			
2	·5	0.2574			
$2\frac{1}{2}$	•4	0.2060			
3 3½	• 3 33 3	0.1716			
$3\frac{1}{2}$.2857	0.1471			
4	.250	0.1287			
4½ 5 6	.2222	0.1144			
5	.2	0.1030			
6	,1666	0.0858			
7 8	.1428	0. 073 5			
8 ,	.125	0.0643			
9	.IIII	0.0572			
10	.10	0.0515			
12	.0833	0.0429			
16	.0625	0.0322			
20	.050	0.0257			

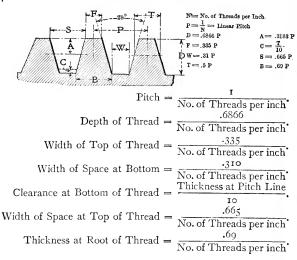


TABLE OF BROWN & SHARPE 29° WORM THREAD PARTS

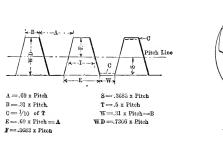
	P	D	F	w	Т	A	С	s	В
Number of Threads Per Inch	Pitch of Single Thread	Depth of Thread	Width of Top of Thread	Width of Space at Bottom	Thickness of Thread at Pitch Line	Thread Above Pitch Line	Clearance at Bottom of Thread	Width of Space at Top	Thickness at Root of Thread
I	1.0	.6866	.3350	.3100	.5000	.3183	.05	.665	.69
$1\frac{1}{4}$ $1\frac{1}{2}$.8	.5492	.2680	.2480	.4000	.2546	.04	.532	-552
$1\frac{1}{2}$.6666	.4577	.2233	.2066	•3333	.2122	.0333	•4433	·459 9
2	-5	•3433	.1675	.1550	.2500	.1592	.0250	-3325	•345
$2\frac{1}{2}$	•4	.2746	.1340	.1240	.2000	.1273	.0200	.2660	.276
3	•3333	.2289	.1117	.1033	.1666	.1061	.0166	.2216	.2299
$3\frac{1}{2}$.2857	.1962	.0957	.0886	.1429	.0909	.0143	.1901	.20II
4	.250	.1716	.0838	.0775	.1250	.0796	.0125	.1637	.1725
$4\frac{1}{2}$.2222	.1526	.0744	.0689	.IIII	.0707	.0111	.1478	.1533
5 6	.2	.1373	.0670	.0620	.1000	.0637	.0100	.1330	.138
	.1666	.1144	.0558	.0517	.0833	.0531	.0083	.1108	.115
7 8	.1428	.0981	.0479	.0443	.0714	.0455	.0071	.095	.0985
8	.125	.0858	.0419	.0388	.0625	.0398	.0062	.0818	.0862
9	.IIII	.0763	.0372	.0344	.0555	.0354	.0055	.0739	.0766
10	.10	.0687	.0335	.0310	.0500	.0318	.005	.0665	.069
12	.0833	.0572	.0279	.0258	.0416	.0265	.0042	.0551	.0575
16	.0625	.0429	.0209	.0194	.0312	.0199	.0031	.0409	.0431
20	.050	.0343	.0167	.0155	.0250	.0159	.0025	.0332	.0345

WORM WHEEL HOBS

Hobs are made larger in diameter than the worm they are used with by the amount of two clearances. The Brown & Sharpe method is to make the clearance one-tenth of the thickness of the tooth on the pitch line or .05 inch for a worm of one pitch. If the worm was 3 inches outside diameter, which would be a fair proportion for this pitch, the outside diameter of the hob would be $3 + (2 \times .05) = 3.1$ inches. The thread tool would be .31 inch wide at the point and would cut .6366 + .r = .7366 deep, leaving the top of the thread the same thickness as the bottom, which is different from the worm.

The land L should be made as near the proportions given as pos-

sible.



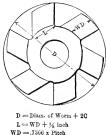


Fig. 18. — Section of Hob Thread Fig. 19. — End View of Hob

The diagram Fig. 18 shows the shape and proportions of the thread of a worm hob, and Fig. 19 shows the proportions for the depth of tooth, the lead and the outside diameter. In these diagrams:

A =Width of space at top of tooth.

B = Width of thread at top.

C =Clearance or difference between the hob and worm.

D = Diameter of hob.

E =Width of tooth at bottom.

F =Hight above pitch line:

L = Width of land or tooth at bottom.

S = Depth below pitch line.

T =Width at pitch line.

W = Width of space at bottom.

WD = Whole depth of tooth.

Brown & Sharpe allow clearance at the point of the tooth only for worm wheels, but at both point and bottom when hobbing spur gears.

PIPE AND PIPE THREADS

BRIGGS STANDARD PIPE THREADS

THE particulars in the following paragraph regarding this system of pipe standards are from a paper by the late Robert Briggs, C.E., read in 1882, before the Institution of Civil Engineers of Great Britain.

The taper employed has an inclination to r in 32 to the axis. The thread employed has an angle of 60 degrees; it is slightly rounded off, both at the top and at the bottom, so that the hight or depth of the thread, instead of being exactly equal to the pitch \times .866 is

only four-fifths of the pitch, or equal to $0.8\frac{1}{n}$, if n be the number of

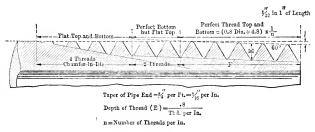


Fig. 1. - Longitudinal Section of Briggs Pipe Thread

threads per inch. For the length of tube-end throughout which the screw-thread continues perfect, the formula used is $(0.8\ D+4.8)$ $\times \frac{1}{n}$, where D is the actual external diameter of the tube throughout its parallel length, and is expressed in inches. Further back, beyond the perfect threads, come two having the same taper at the bottom, but imperfect at the top. The remaining imperfect portion of the screw-thread, furthest back from the extremity of the tube, is not essential in any way to this system of joint; and its imperfection is simply incidental to the process of cutting the thread at a single operation.

Thread Section

The threads as produced at the pipe end in the Briggs system are represented clearly in the longitudinal section, Fig. 1.

Here the threads that are perfect at top and bottom are shown at F, the depth being indicated at E. Back of the perfect threads

are represented the two threads with perfect bottom and flat tops and behind these are the imperfect threads produced by the chamfer or bell mouth of the threading die. A table giving the general dimensions of wrought iron tubes in the Briggs system will be found on page 40, while complete data pertaining to the thread depths, lengths of perfect and imperfect portions, allowances for making the joint in screwing the pipe into the fitting, gaging allowances, etc., are contained in the tables on pages 47 and 48.

In cutting pipe threads with a lathe tool as in threading taper work in general, the tool should be set at right angles to the axis

of the piece and not square with the conical surface.

STANDARD DIMENSIONS OF WROUGHT-IRON WELDED TUBES
BRIGGS STANDARD

Di	ameter of Tul	oes	Thickness	Screwed	Ends
Nominal Inside Inches	Actual Inside Inches	Actual Outside Inches	of Metal Inches	Number of Threads per Inch	Length of Perfect Thread Inches
1 14-12 2 3 3 4 1 5 6 7 8 9 10	0.270 0.364 0.494 0.623 0.824 1.048 1.380 1.610 2.067 2.468 3.067 3.548 4.026 4.508 5.045 6.065 7.023 7.982 9.000	0.405 0.540 0.675 0.840 1.050 1.315 1.660 1.900 2.375 2.875 3.500 4.500 5.563 6.625 7.625 8.625 9.688	0.068 0.088 0.091 0.109 0.113 0.134 0.145 0.154 0.204 0.217 0.226 0.237 0.246 0.259 0.280 0.301 0.322 0.344 0.366	27 18 18 14 14 11 12 1-12 11 12 8 8 8 8 8 8 8 8 8	0.19 0.29 0.39 0.40 0.51 0.54 0.55 0.89 0.95 1.00 1.05 1.16 1.26 1.36 1.46 1.57

^{*}By the action of the manufacturers of wrought-iron pipe and boiler tubes, at a meeting held in New York, May 9, 1859, a change in size of actual outside diameter of 9-inch pipe was adopted, making the latter 9.625 instead of 9.688 inches, as given in the table of Briggs standard pipe diameters.

 $T_{\rm HE}$ table below shows the British pipe and pipe threads, sizes recommended by the Engineering Standards Committee.

BRITISH STANDARD PIPE THREADS

	DRITISH STANDARD THE THREADS												
)iameter ipe	ate Out- imeter ipe	ımeter, Fhread	Depth read	Diameter Bad at Point	f Thread nch	DIA	ANCE OI METER ND OF F	FROM	Diameter ads at f Pipe	iameter ead at f Pipe			
. Nominal Diameter of Pipe	Approximate Outside Diameter of Pipe	Gage Diameter, Top of Thread	Double Depth of Thread	Bottom Diameter of Thread at Gage Point	Number of Thread per Inch	Standard	Maxi- mum	Mini- mum	Outside Diameter of Threads at End of Pipe	Inside Diameter of Thread at End of Pipe			
$\frac{1}{8}$	13 32	0.383	0.046	0.337	28	35	0.18	0.13	0.373	0.327			
1	17 32	0.518	0.067	0.451	19	3 1 6	0.22	0.16	0.506	0.439			
3	11	0.656	0.067	0.589	19	1 4	0.29	0.21	0.640	0.573			
14 30 12 50 34 70	27	0.825	0.091	0.734	1.4	14	0.29	0.21	0.809	0.718			
5 8	15	0.902	0.091	0.811	14	14	0.29	0.21	0.886	0.795			
3	$\mathbf{I}_{\mathbf{T}^{i}}^{1}$	1.041	0.001	0.950	1.4	3 8	0.44	0.31	1.018	0.927			
7 8	I 37	1.189	0.001	1.098	14	38	0.44	0.31	1.166	1.075			
1	111	1.309	0.116	1.193	11		0.44	0.31	1.286	1.170			
I 1/4	111	1.650	0.116	1.534	11	$\frac{1}{2}$	0.58	0.42	1.619	1.503			
I $\frac{1}{2}$	I 3 9	1.882	0.116	1.766	11	$\frac{1}{2}$	0.58	0.42	1.851	1.735			
$1\frac{3}{4}$	2 3 2	2.116	0.116	2.000	11	5 8	0.73	0.52	2.077	1.961			
2	2 3	2.347	0.116	2.231	II	ş	0.73	0.52	2.308	2.192			
$2\frac{1}{4}$	2 5/8	2.587	0.116	2.471	11	11	0.80	0.57	2.544	2.801			
$2\frac{1}{2}$	3	2.960	0.116	2.844	11	11	0.80	0.57	2.917	3.043			
$2\frac{3}{4}$	3 1/4	3.210	0.116	3.094	ΙI	13	0.95	0.68	3.159	3.293			
3	3 1/2	3.460	0.116	3.344	11	13	0.95	0.68	3.409	3.293			
31/4	3 3	3.700	0.116	3.584	11	7 8	1.02	0.73	3.645	3.529			
3 2	4	3.950	0.116	3.834	11	7 8	1.02	2.73	3.895	3.779			
$3\frac{3}{4}$	4 4	4.200	0.116	4.084	11	7 8	1.02	0.73	4.145	4.029			
4	4 1/2	4.450	0.116	4.334	II	I	1.17	0.83	4.387	4.271			
$4\frac{1}{2}$	5	4.950	0.116	4.834	II	I	1.17	0.83	4.887	4.77I			
5	5 1/2	5.450	0.116	5.334	II	I 1/8	1.31	0.94	5.380 5.872	5.264 5.756			
5 1/2	6	5.950	0.116	5.834	11	I 1/4 I 3/8	1.46	1.04	6.364	6.248			
6	6 1/2	6.450	0.116	6.334	11	1 8 1 3	1.60	1.15	7.364	7.236			
7 8	7 ½ 8 ½	7.450	0.128	7.322	10	1 ½	1.75	1.15	8.356	8.228			
	$0 \frac{1}{2}$	8.450	0.128	8.322 9.322	10	I 1/2	1.75	1.25	9.356	9.228			
9	10 1/2	9.450	0.128	10.322	10	I 5	1.90	1.35	10.348	10.220			
11	11 1/2	11.450	0.160	11.290	8	I 5	1.90	1.35	11.348	11.188			
12	12 1/2	12.450	0.160	12.290	8	I 5	1.90	1.35	12.348	12.188			
13	13 3	13.680	0.160	13.520	8	I 5	1.90	1.35	13.578	13.418			
14	14 3	14.680	0.160	14.520	8	I 3	2.04	1.46	14.571	14.411			
15	15 3	15.680	0.160	15.520	8	I 3	2.04	1.46	15.571	15.411			
16	16 3	16.680	0.160	16.520	8	1 7	2.19	1.56	16.563	16.403			
17	17 3	17.680	0.160	17.520	8	2	2.33	1.67	17.555	17.395			
18	18 3	18.68o	0.160	18.520	8	2	2.33	1.67	18.555	18.395			
	1			-	1		l						

TAP DRILLS FOR PIPE TAPS

The sizes of Twist Drills to be used in boring holes, to be reamed with Pipe Reamers, and Threaded with Pipe Taps, are as follows:

Size	BRIG	GS	WHITWO	ORTH	Size Pipe	BRIG	GS	WHITWORTH		
Pipe Tap	Thread	Tar		Tap	Thread	Drill	Thread	Drill		
18	27	2 1 6 4	28	1 ⁵ 6	1 3/4			II	I 15	
$\frac{1}{4}$	18	$\frac{27}{64}$	19	$\frac{27}{64}$	2	112	$2\frac{3}{16}$	II	$2\frac{5}{32}$	
1 4 3 8	18	9 16	19	$\frac{9}{16}$	$2\frac{1}{4}$			11	$2\frac{1}{3}\frac{3}{2}$	
$\frac{1}{2}$	14	$\frac{11}{16}$	14	$\frac{1}{1}\frac{1}{6}$	$2\frac{1}{2}$	8	$2\frac{9}{16}$	II	$2\frac{2}{3}\frac{5}{2}$	
<u>5</u>			14	$\frac{25}{32}$	$2\frac{3}{4}$			ΙI	3	
12 58 34 78	14	$\frac{29}{32}$. 14	$\frac{29}{32}$	-3	8	$3\frac{3}{16}$	11	3 1/4	
7 8			14	$1\frac{1}{16}$	31/4			11	$3^{\frac{1}{2}}$	
1	$\mathbf{I} \mathbf{I} \frac{1}{2}$	I 1/8	II	I 1/8	$3\frac{1}{2}$	8	$3\frac{1}{1}\frac{1}{6}$	11	3 3/4	
$I_{\frac{1}{4}}$	$II\frac{1}{2}$	$1\frac{1}{3}\frac{5}{2}$	ΙΙ	$1\frac{1}{3}\frac{5}{2}$	3 4			11	4	
$I^{\frac{1}{2}}$	1112	$1\frac{2}{3}\frac{3}{2}$	11	$1\frac{2}{3}\frac{3}{2}$	4	8	$4\frac{3}{16}$	II	$4\frac{1}{4}$	
				1	1					

METRIC PIPE THREADS

Nominal Inside Pipe Diameter in Inches	Inside Pipe Diameter in Millimeters	External Thread Diameter in Millimeters	Internal Thread Diameter in Millimeters	Number of Threads per Inch
<u>1</u> 8		10	8.3	19
1/4	6.35	13	11.3	19
ৰতি মধি লাভ মাথ চাত লাধা	9.52	16.5	14.8	19
$\frac{1}{2}$	12.70	20.5	18.2	14
<u>5</u> 8	15.87	23	20.7	14
34	19.05	26.5	24.2	14
1	25.40	33	30	11
$1\frac{1}{4}$	31.75	42	39	II
$I\frac{1}{2}$	38.10	48	45	II
I 3/4	44.45	52	49	11
2	50.30	59.7	56.7	II
$2\frac{1}{2}$	63.50	76	73	II
3	76.20	89	86	11
$3\frac{1}{2}$	88.90	101.5	98.5	II
4	101.60	114	111	11

THE PIPE JOINT IN THE BRIGGS SYSTEM

THE illustrations below and the tables on pages 43 and 44, represent the relation of the reamer, tap, die and testing gages in the preparation of the Briggs pipe end and fitting preliminary to making up the joint.

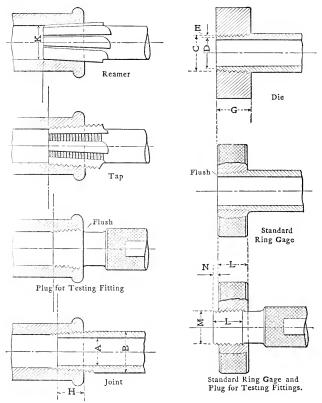


Fig. 2. — Reamer, Tap, Die and Gages for Briggs Pipe Standard

The illustrations to the left in Fig. 2 show the relative distances that the pipe reamer, tap, testing plug and pipe end are run into the fitting in making the joint; while at the right are shown the die and ring gage on the pipe end, and the relative diameters of the standard ring gage and the testing plug for the fittings.

In pipe fitting the end of the pipe should always be cut to fit the Briggs standard pipe gage. The fitting should be tapped small in order to insure a tight joint. Theoretically the joint should be tight when the pipe end has been screwed into the fitting a distance represented at H in the diagrams, Fig. 2 and following tables. However, to allow for errors the thread on the pipe is actually cut two threads beyond H. Similarly the fitting should be tapped two threads deeper than distance H.

The following table used in conjunction with the illustrations in Fig. 2, contains information as to length and number of perfect and imperfect threads; distance and number of turns the pipe screws into fitting by hand and with wrench, or the total length and number of threads of joint; ring and plug gage data for testing tools;

besides general pipe dimensions, drill and reamer sizes, etc.

BRIGGS PIPE THREAD TABLE (See page 46)

Dia. o Nominal Inside	Actual Inside	Actual Outside	No. of Threads per Inch	Dia. at End of Pipe	Dia. at Bottom of Th'd	Depth of Thread	Length of Perfect Threads	No. of Perfect Threads
	A	В		C	D	Е	Н.	
10 14 clar of class o	.270" .364" .494" .623" .824" 1.048" 1.380" 1.610" 2.067" 3.548" 4.026" 4.508" 7.023" 7.982" 9.000"	.840" I.050" I.315" I.660" I.900" 2.375" 2.875" 3.500" 4.000" 5.563" 6.625" 8.625"	18 14 11 ¹ / ₂ 11 ¹ / ₂ 11 ¹ / ₂ 8 8 8 8 8 8 8	.393", .522", .656", .815", 1.025", 1.836", 2.339", 2.819", 3.441", 3.938", 4.434", 4.931", 5.490", 7.540", 8.534", 9.527", 10.645"		.069" .069" .100" .100" .100" .100" .100" .100"	.19" .29" .30" .30" .51" .55" .58" .58" 1.00" 1.16" 1.16" 1.36" 1.46" 1.57" 1.68"	5.13 5.22 5.4 5.46 5.6 5.87 6.21 6.33 6.67 7.12 7.6 8.0 8.4 8.8 9.28 10.08 10.88 11.68 12.56 13.44

	indituo.
E	ARIT
E	HREAD
	LIDE
ſ	X D L C L C L

	Dia. of End of Pipe Reamer Before Chamfering	M	.320"	.414"	.539"	.648"	.867	"IOI.I	1.449"	1.695"	2.164"	2.554"	3.180"	3.680″	4.180″							
	Dia. of Drill to be used with Pipe Reamer		: NO	아(O	1.6 1.0	10	≥ ⊕ ⊕ (0)(0)	I 3 "	135" 135"	I 2500	2 1 6	216	316"	316"	416							_
	Distance In- spection Plug Pro- stoets tini King	Z	"OIII.	.1805"	.1805"	.250"	.250"	.304"	.304"	.304″	.348"	.500	.5625"	.5625	.5025	.5625"	.625	.625	.6125	.6495	.6825	.7150"
40)	No. of Th'ds Inspection Plug Pro- jects through Ring		3	3.1	34	31	$3\frac{1}{2}$	32	3 2	32	4	4	42	42.	421	42	ις	Ŋ	4.9	5.196	5.46	5.72
(See page	Dia. of End of Plug for Testing Fit- tings	M	.3861"	.5107"	.6447	.7994	1.0004"	1.2640"	1.607"	1.847"	2.3173"	2.787 "	3.4058"	3.9028"	4.3988"	4.8958"	5.4509"	6.5069"	7.5017"	8.4934"	9.4843"	10.6003"
— Continued.	Length of Thread on Plug and Ring Gage	T	.2638"	.4018	.4078″	.5337"	.5457"	.6828"	.2002.	.7235"	.7565"	1.1375"	1.2000"	1.2500"	1.3000"	1.3500"	1.4063"	1.5125"	1.6125"	1.7125"	1.8125"	1.9250"
TABLE — C	No. of Th'ds on Plug and King Gage		7.1256	7.2324	7.3404	7.4718	7.6398	7.8522	8.1282	8.3203	8.6998	9.1	9.6	10.0	10.4	10.8	11.2504	12.1	12.9	13.7	14.5	15.4
THREAD 1	Amount of "Take Up" with Wrench		"01111.	.1805"	.1805"	.250″	.250″	.304"	.304"	.304″	.348"	.500	.5625"	.5625"	.5625"	.5625"	.625 "	.625	.6125"	.6495"	.6825"	.7150"
PIPE T	No. of Turns "Take Up" with Wrench		8	3 1	34	32	32	32	3,1	31	4	4	421	421	4212	4	v	Ŋ	4.9	5.196	5.46	5.72
BRIGGS	Distance Pipe Screws into Fitting by Mand		.1528"	.2213"	.2273"	.2837"	.2957	.3788"	.4028"	.4195"	.4085"	.6375"	.6375"	.6875	.7375"	.7875"	.7813"	.8875	1.0000	1.0630"	1.1300"	1.2100"
	No. of Turns Pipe Screws into Fitting banh yd		4.1256	3.9834	4.0914	3.9718	4.1398	4.3562	4.6322	4.8243	4.6978	5.1000	5.1000	5.5000	5.9000	6.3000	6.2504	7.1000	8.0000	8.5040	9.0400	0089.6
	Total Length of Th'd and Thickness of Thickness of Dickness	g	.4120"	.6240"	.6300"	.8194"	.8315"	1.0305"	1.0545"	1.0713"	1.1043"	1.6375"	1.7000"	1.7500"	1.8000"	1.8500"	1.9063"	2.0125"	2.1125"	2.2125"	2.3125"	2.4250"
	Nominal Dia. of Pipe		* = ×	, T	<u>*</u>	= - (0)	E 4	" I	$1\frac{1}{4}''$	$\Gamma_2^{\frac{1}{2}''}$	20	2 3 "	3 "	32"	4	42 / /	25	, 9	, , ,	* 8	, 6	" OI

GAGE SETS FOR BRIGGS PIPE AND FITTINGS

The gages manufactured by the Pratt & Whitney Company for makers and users of pipe and fittings include three distinct sets for each size of pipe, and these are illustrated in Fig. 3. Set No. 1 consists of a ring and plug conforming in all dimensions to the Briggs standard, and is known as the standard reference set. The plug screws into the ring with faces flush—as indicated by the position of the two gages. The flat milled on the plug shows the depth to which the latter should enter the fitting to allow for screwing up with tongs to make a steam-tight joint; the ring, of course, screws on to the pipe flush with the end.

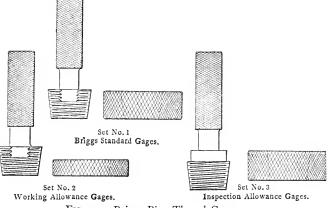


Fig. 3. — Briggs Pipe Thread Gages

Set No. 2—the working allowance set—consists of the plug already described and a ring whose thickness is equal to the standard ring less the allowance for screwing up the joint. As the plug and ring threads are of the same diameter at the small end, the bottom surfaces come flush when the two members are screwed together. It will be noted that, as the plug enters the fitting only to the bottom of the flat at the side, and the ring screws on to the pipe only far enough to bring the outer face flush with the pipe end, there are a few threads on, or in, the work beyond the reach of the gages; hence with this type of gage a reasonable amount of wear may be permitted at the end of the tap or the mouth of the die without causing the rejection of the work.

The plug and ring in set No. 3 are inspection allowance gages, the ring being the same in all particulars as the standard gage in set No. 1, while the plug is longer than Nos. 1 and 2 by an amount equal to the allowance for screwing up for a tight joint, this extra length being represented by the cylindrical portion at the rear of

the thread cone. When the gages are screwed together the back of the cylindrical section comes flush with the ring face and the threaded end of the plug projects through the ring, as indicated, a distance equal to the length of the cylinder, or the screwing-up allowance. This plug will enter a perfect fitting until the back of the threaded section is flush with the end of the fitting, thus testing the full depth of the tapped thread in the same way that the standard ring gage covers the thread on the pipe end, and at the same time showing that the fitting is tapped to right diameter to allow the joint to screw up properly.

NATIONAL STANDARD HOSE COUPLING

This standard for fire hose couplings was adopted by the National Fire Protection Association May 26, 1905 and has since been approved and adopted by various other organizations.

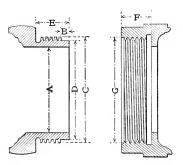


Fig. 4. - National Standard Hose Coupling

DIMENSIONS OF NATIONAL STANDARD HOSE COUPLINGS

A B C D E	Inside Diameter of Hose Couplings Length of Blank End on Male Part Outside Diameter of Thread, Finished Diameter at Root of Thread Total Length of Male End	Ĭ.	3 35 35 3.3763	3½ 4 44 4.0013	4 ¹ / ₂ 5 ³ / ₄ 5.3970
F G	Number of Threads per inch Length of Female Thread Diameter of Top of Female Thread	7 8	6 1 3.6550	6 1 4.28	4 114 5.80

Note:—The above to be of the 60-deg. V-thread pattern with one-hundredth inch cut off the top of thread and one-hundredth inch left in the bottom of the $2\frac{1}{2}$ -inch, 3-inch, and $3\frac{1}{2}$ -inch couplings, and two hundredths inch in like manner for the $4\frac{1}{2}$ -inch couplings, and with one-quarter inch blank end on male part of coupling in each case; female ends to be cut $\frac{1}{2}$ -inch slorter for endwise clearance. They should also be bored out .03 inch larger in the $2\frac{1}{2}$ -inch, 3 and $3\frac{1}{2}$ -inch sizes, and .05 inch larger on the $4\frac{1}{2}$ -inch size in order to make up easily and without jamming or sticking.

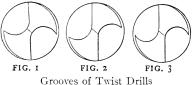
TWIST DRILLS

THE twist-drill is perhaps one of the most efficient tools in use as, although one half is cut away in the flutes, it has a very large cutting surface in proportion to its cross-sectional area. This is made possible by the fact that the work helps to support the drill and the feed pressure on the drill tends to force the point into a cone-shaped hole which centers it.

In addition to the radial relief or backing-off behind the cutting edge, twist-drills have longitudinal clearance by decreasing the diameter from the point toward the shank, varying from .00025 to .0015 per inch of length. This prevents binding and is essential in accurate

drilling.

To increase the strength the web is increased gradually in thickness from the point toward the shank by drawing the cutters apart. This decreases chip room and to avoid this defect the spiral is increased in pitch and the flute widened to make up the chip room.



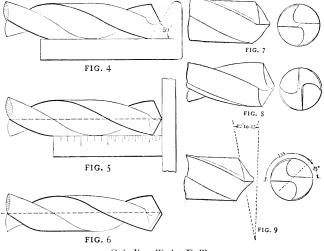
The shape of the groove affects the power and the shape of the chip and experiments by the Cleveland Twist-Drill Company are The groove in Fig. 1 does not give a good cutting edge, especially near the center, as it does not allow a full curl to the chip. Fig. 2 is a very free cutting-groove, the chips curl up to the full size of the groove and this reduces the power required to bend the cnips. Fig. 3 is an even better form as it rolls a chip with each turn conical so that one lays inside the other and makes a much shorter chip from the same depth of hole.

The angle of spirals varies from 18 to 35 degrees according to the ideas of the maker. In theory the finer the pitch or the greater the angle, the easier it should be to cut and curl the chip. But this gives a weak cutting edge and reduces the ability to carry off the heat, and it does not clear itself of chips so well. After a long series of tests the same firm adopted 27½ degrees for the spiral. angle makes the spiral groove of all drills start at the point with a pitch equal to six diameters of the blank, the increase in twist being a constant function of the angular movement of rotation of the drill blank. This angle is based on holes from one to three diameters deep. For deeper holes a smaller angle might be advisable and greater angle for holes of less depth. There is practically no difference in torsional stress with the angle between 25 and 30 degrees.

SHARPENING DRILLS

Drills should be sharpened so as to cut the right size and with as little power as possible. To cut the right size both lips must be the same length and the same angle. A gage as shown in Fig. 4 will help both to get the angle and to grind them central. This gives the usual lip edge of 59 degrees. Fig. 5 shows how you can see if both lips are ground alike, but does not give the angle. Fig. 6 is a suggestion by Professor Sweet of relieving the drill back of the cutting edge, making it similar to a flat drill in this respect.

For drilling brass or for any thin stock where the drill goes clear through, it is best to grind the cutting edge parallel with the axis of



Grinding Twist Drills

the drill. This does away with the tendency to draw into the work. Fig. 7 shows how this is done.

It is sometimes necessary to thin the point of the drill to get best results. This requires care in grinding but can be done as shown in Fig. 8.

The best all-around clearance angle is 12 degrees, though for softer metals 15 degrees can be used. The 12 degrees is the angle at the cutting edge, but this should increase back of the cutting edge so that the line across the web should be 45 degrees, with the cutting edges. This is important, as it not only saves power but prevents splitting in hard service. The point of the drill should look like Fig. 7 or Fig. 8. Fig. 9 shows the clearance angle and the right angles for the drill point.

SPEED OF DRILLS

Learn to run drills at their proper speed to secure the most work with fewest grindings and breakages. The best practise is to use a speed that will give 30 feet a minute cutting speed for steel, 35 feet for cast iron and 60 feet for brass. This means that the cutting edge must run fast enough to make these speeds. For drilling steel with a ½-inch drill this means 1834 revolutions a minute, while for brass it would be 3668 revolutions. The table gives the speeds without any figuring for all drills up to 3 inches. These speeds require plenty of lubricant. This is for carbon steel drills.

These speeds can be exceeded in many cases even with carbon drills, and can be doubled with high speed drills, in fact from 75 to 150 feet is not uncommon with 200 feet a possibility under good conditions.

The feeds in the table below can also be doubled in many cases.

TABLE OF DRILL FEEDS

=	-	Inches	of Fee	d per Min	ute at C	Cutting	Speed of			
Diam. of Drill	30	Feet-Steel		35 F	`eet-Iro	n	60 Feet-Brass			
Diam.	Rev. per Minute	Feed .oo4	007	Rev. per Minute .004007			Rev. per Minute	.004007 per Revolution		
1 16 18 3 16 14	1834 917 611 458	7·33 3.66 2·44 1.83 Feed .007	12.83 6.41 4.27 3.20 .015	2140 1070 713 535	8.56 4.28 2.85 2.14 .007	14.97 7.49 4.99 <u>3.74</u> .015	3668 1834 1222 917	14.66 7.33 4.88 3.66 .007	25.76 12.83 8.58 6.44 .015	
5168776121588478 I I I I I I I I 2 2 2 2 2	367 306 262 229 184 153 131 1102 91.8 83.3 76.3 70.5 65.5 61.1 57.3 41.7	.42 .40 .36	5.5 4.6 3.9 3.43 2.75 2.3 1.95 1.71 1.53 1.37 1.25 1.15 1.05 .97 .92 .85 .71	97.2 89.2 82.2 76.4 71.3 66.9 59.4 53.5	3 2.5 2.14 1.87 1.50 1.25 1.07 .93 .83 .75 .68 .62 .57 .53 .50 .40	6.42 5.35 4.58 4. 3.21 2.67 2.29 2 1.79 1.61 1.45 1.38 1.15 1.07 1.	611 524 459 367 306 262 229 204 183 167 153 141 131 122 115 102 91.7	5.14 4.28 3.66 3.20 2.57 2.14 1.88 1.60 1.43 1.17 1.07 .99 .85 .80 .71 .64 .58	9.2 7.8 6.86 5.5 4.6 3.93 3.06 2.75 2.51 2.3 2.11 1.73 1.53 1.53 1.21	
3	38.2		-57	1	.31	.67			1.15	

FEED OF DRILLS

The feed of drills is usually given in parts of an inch per revolution, 0.004 to 0.007 inch for drills of $\frac{1}{4}$ inch and smaller and 0.007 to 0.015 inch for larger drills being recommended. This has been worked out into the table for the standard speeds to show inches of feed per minute for the three speeds given, which is more convenient. This is not an iron-clad rule but should be used with judgment. For high-speed steel these figures can be just about doubled.

DATA FOR DRILLING CAST IRON AT FEED OF I" PER MIN.

Size of Drill	Feed per Rev.	Thrust in Lbs.	H.P. for 1 Inch Feed per Min.	Size of Drill	Feed per Rev.	Thrust in Lbs.	H.P. for 1 Inch Feed per Min.
I	.02	1300	.0035		.06	8000	.02
	.04	2600	.0063	$2\frac{1}{2}$.02	3200	.008
	.06	3900	.010		.04	6500	.016
$\mathbf{I}^{\frac{1}{2}}$.02	2000	.005		.06	9700	.024
- 1	.04	3900	.010	3	.02	3750	.009
	.06	5800	.015		.04	7700	.019
2	.02	2500	.006		.06	11500	.029
	.04	5300	.013				

For carbon steel the values run from $1\frac{1}{2}$ to 3 times these for cast iron, increasing with the feed per revolution.

One inch flat twisted drills have been run from 313 to 575 r. p. m., with feeds of 11.27 and 28.1 inches per minute and required from 5.22 to 11.60 actual horse power.

Torque Required to Drill Cast Iron

Diam. of Drill	Feed in Inch per Rev.	Pounds Torque at 1 foot Radius	H.P. per Rev.	Diam of Drill	Feed in Inch per Rev.	Pounds Torque at 1 foot Radius	H.P. per Rev.
I I ½ 2	.02 .04 .06 .02 .04 .06 .02	50 80 120 75 150 225 125	.009 .014 .023 .014 .028 .042 .023	$2\frac{1}{2}$ 3	.06 .02 .04 .06 .02 .04	390 200 400 600 280 575 870	.072 .038 .076 .114 .053 .109

DRILL TROUBLES

Twist-drills will stand more strain in proportion to their size and weight than almost any other tool, and when a good drill gives trouble it is pretty safe to say some of the conditions are wrong.

If it chips on the edge, the lip clearance is too great and fails to support the cutting edge or the feed is too heavy. Ease off on the

feed first and then watch the grinding.

If it splits in the web it is either ground wrong, i.e., does not have the center lip at the angle of 45 degrees or the feed is altogether too heavy.

If the outer corner wears, it shows that the speed is too great.

This is particularly noticeable on cast iron.

Drill Pointers

In most cases it is better to use high speeds almost to the point where the drill corners commence to wear with a light feed than to

use slower speed and heavy feed.

This is specially true of drilling in automatic machines where the holes are not more than twice as deep as the diameter where drills are flooded with lard oil. With deeper holes the chips are harder to get rid of and it is better to use slower speeds and heavier feeds as the drilled hole gets deeper. Speeds of 10,000 r. p. m. for drills $\frac{1}{16}$ in. and smaller are not uncommon.

Watch the drill chip and try to grind so that it will come out in a small compact roll. It is better to have this continuous clear to

bottom of hole if possible.

In drilling brass use a heavier feed especially on automatic machines, as it helps to work out the chips. If you lubricate at all, flood the work. Twist-drills ground as for steel often catch and "hog in" on brass, especially at the bottom of the hole, where it breaks through. To avoid this, grind the lead or rake from the cutting edge.

In drilling hard material use turpentine as a lubricant.

Drills feed easier by thinning the extreme point if this is carefully

done. This is important in hand feeding.

High-speed drills work best when warm. Lubricant should be heated to 150 degrees F. when starting drills to work; they will soon maintain the proper temperature.

Special Drills and Their Uses

Ratchet-drills have a square taper shank, are used in hand-ratchet braces and in air-driven drills. Used in bridge building, structural

and repair work.

The shell drill, Fig. 10, is used after a two-groove drill in chucking out cored holes or for enlarging holes that have been made with a two-groove drill. It has a taper hole and a number of sizes can be used on the same arbor.

Wire drills and jobbers or machinists drills both have round shanks and only differ in size. Wire drills are made to a twist-drill gage and the others to a jobbers or fractional gage.

Blacksmith drills all have a ½-inch shank 2½ inches long, so as to all fit the same holder. There is a flat on the shank for set-screw.

The straightway or Farmers drill has the same clearance as a twist-drill but the flutes are straight. It is used mostly in drilling brass and soft metals or in drilling cross holes or castings where blow holes may be found, as it is less likely to run than the twist-drill.

Oil drills have the advantage of the cutting edge being kept cool and of the chips being forced back through the grooves which reduces friction to a minimum. They are used for all kinds of drilling, mostly deep hole work. In cast-iron drilling air is sometimes used to blow out the chips and keep the drill cool. They are generally used in a screw or chucking machine or a lathe fitted for this work. Where the drill is held stationary and the work revolves, the oil is pumped to the connection and flows through the holes in the drill

as in Fig. 11.

Where the drill revolves as in a drill press, the oil is pumped into a collar which remains stationary while the drill socket revolves, as in Fig. 12. An oil groove around the socket and holes through to the drill connects with the holes in the drill itself. Other types are shown in Figs. 13 and 14. The latter is used mostly

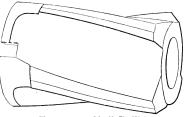


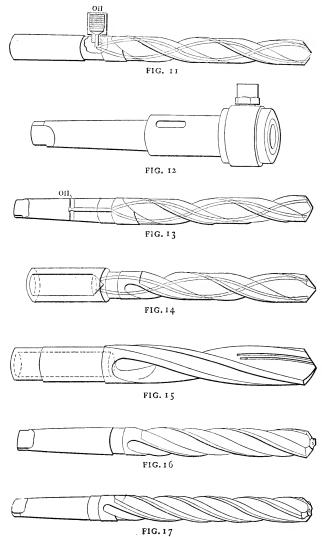
Fig. 10. - Shell Drill

in screw or chucking machine turrets where the oil is pumped into the center of the turret and into the large hole in the shank of the drill.

The hollow drill shown in Fig. 15 is used for deep drilling or long holes and is used in a lathe or some similar machine fitted for the purpose. It has a hole lengthwise through the shank connecting with the grooves of the drill. The shank can be threaded and fitted to a metal tube of such length as desired. The outside of the drill has a groove the whole length of the body. The lubricant is conveyed to the point of the drill on the outside through these grooves, while the hollow tube admits of the passage of oil and chips from the point. In using this drill the hole is first started with a short drill the size of the hole desired and drilled to a depth equal to the length of the body of the hollow drill acts as a packing, compelling the oil to follow the grooves and the chips to flow out through the hollow shank.

Three and four groove drills are used for chucking out cored holes or enlarged holes that are first drilled with a two-groove drill. They are much better than a two-groove drill for use in cored holes or to follow another drill. The ends of the drills, Fig. 16 and 17, indicate that they are not made to drill from solid stock but for enlarging a

hole already made



The following tables give standard drill sizes in various ways, each being very convenient for certain classes of work:

DECIMAL EQUIVALENTS OF NOMINAL SIZES OF DRILLS

Inch	M.M.	Wire Gage	Decimals of an Inch	Inch	M.M.	Wire Gage	Decimals of an Inch	Inch	M.M.	Wire Gage	Decimals of an Inch
1 64	•4	80 79 78 77	.0135 .0145 .015625 .01574 .016		1.2 1.3	55 54	.047244 .051181 .052 .055 .055118	e ⁷ 4	2.7	37 36 35	.104 .1063 .1065 .109375 .11
	.6	76 75 74	.01968 .020 .021 .0225 .02362	1 16	1.6	53 52	.0595 .0625 .06299 .0635 .066929		2.9	34 33 32	.111 .113 .11417 .116 .11811
	.7	73 72 71 70 69	.024 .025 .026 .02756 .028		1.8	51 50 49 48	.067 .07 .070866 .073 .0748	18	3.1 3.2 3.3	30	.12 .12205 .125 .12598 .1285
32	.8	68 67 66	.031 .03125 .031496 .032	5 64	2	47 46 45	.078125 .0785 .07874 .081	9 64	3·4 3·5	29 28	.13386 .136 .1378 .1405
	.9	65 64 63 62	.035 .03543 .036 .037 .038		2.1 2.2 2.3	44 43	.082677 .086 .086614 .089		3.6 3.7	27 26 25	.14173 .144 .14567 .147
	I	60 59 58	.039 .03937 .04 .041	3 2	2.4	41 40	.0935 .09375 .09448 .096	5 32	3.8 3.9	24	.14961 .152 .15354 .154 .15625
<u>8</u> 4	1.1	57 56	.043 .043307 .0465 .046875		2.5	39 38	.098425 .0995 .1015 .102362		4	22 21 20	.157 .15748 .159

DECIMAL EQUIVALENTS OF NOMINAL SIZES OF DRILLS, Continued

-				1	1			1			
Inch	M.M.	Wire Gage	Decimals of an Inch	Inch	M.M.	Letter Sizes	Décimals of an Inch	Inch	M.M.	Letter Sizes	Decimals of an Inch
\$\frac{1}{6}\frac{1}{4}\$	4.1 4.2 4.3 4.4 4.5 4.6 4.7 4.8 4.9 5 5.1 5.2 5.3 5.4 5.5 5.6 5.7 5.8 5.9	19 18 17 16 15 14 13 10 9 8 7 6 5 4 3 2 1	.16142 .16536 .166 .16929 .1695 .171875 .17323 .177 .17717 .18 .1811 .182 .185 .18504 .1875 .18898 .191 .19291 .1935 .196 .19685 .199 .20079 .201 .203125 .204 .20473 .2055 .20866 .213 .21654 .21875 .22047 .221 .228 .22835 .23228	1.7.1 1.7.1 1.7.1 1.7.1 1.7.1 1.7.1 1.7.1	6 6.1 6.2 6.3 6.4 6.5 6.6 6.7 7.1 7.2 7.3 7.4 7.5 7.6 7.7 7.8 7.9 8.	B C D E F G H I J K L M	.234 .234375 .23622 .238 .24015 .242 .2441 .246 .24803 .25 .25197 .25591 .257 .25984 .261 .26377 .265625 .266 .26772 .27165 .277 .27952 .281 .28125 .28347 .2874 .29 .29133 .295 .29528 .296875 .29528 .296875 .302 .30314 .30709 .31102 .3125 .31496 .32284	260 12 2 24 2 24 2 24 2 24 2 24 2 24 2 24	8.5 8.6 9 9.2 9.5 9.6 9.8 10 11.5 11 12 12.5	P Q R S T U V W X Y Z	.323 .328125 .332 .33465 .33859 .339 .34546 .348 .35433 .358 .359375 .368 .37402 .375 .377 .37796 .38583 .390625 .3937 .404 .40625 .413 .4134 .421875 .45276 .453125 .46875 .47244 .484375 .4921
-									•		

DECIMAL EQUIVALENTS OF NOMINAL SIZES OF DRILLS, Continued

Inch M.M.	Decimals of an Inch	Inch	M.M.	Decimals of an Inch	Inch	M.M.	Decimals of an Inch
13. 3. 13. 13. 13. 13. 14. 15. 15. 15. 16. 16. 17. 17. 17. 17. 17. 17. 17. 17. 17. 17	.546875 .55118 .5625 .57087 .578125 .59055 .59375 .609375 .61024 .625 .62992	2)+1-10 5)4 5)1 1-1+ 5)0 3]4 5]+ 5)0 1-1- 5)0 1-1- 5)0 1-	17.5 18 18.5 19 19.5 20 20.5	.671875 .6875 .6895 .703125 .70866 .71875 .72835 .734375 .755 .765625 .76772 .78125 .7874 .796875 .8071 .8125 .82677 .828125	7-22 5-14 7-28 5-16 2505-10 1-14 1-12 5-14 1-1	21.5 22 22.5 23 23.5 24 24.5 25	.84375 .84646 .859375 .86614 .875 .88533 .896625 .90551 .90625 .921875 .9252 .9375 .94488 .953125 .9646 .96875 .98425 .984375

LETTER SIZES OF DRILLS

Diameter	Decimals	Diameter	Decimals
Inches	of 1 Inch	Inches	of 1 Inch
A 154 B C C D E 1 F G H 167 I J K 30 L M 164	.234 .238 .242 .246 .250 .257 .261 .266 .272 .277 .281 .290	$\begin{array}{c} N \\ O \\ \begin{array}{c} \frac{5}{161} \\ P \\ \frac{21}{64} \\ Q \\ R \\ \frac{1}{312} \\ S \\ T \\ \frac{28}{64} \\ U \\ W \\ \frac{3}{6} \\ \frac{5}{4} \\ W \\ \frac{3}{32} \\ Z \\ \end{array}$.302 .316 .323 .332 .339 .348 .358 .368 .377 .386 .397

Decimal Equivalents of Drill Sizes from 1/2" to No. 80

TAP DRILL SIZES FOR REGULAR THREADS

These sizes give an allowance above the bottom of thread on sizes $\frac{3}{16}$ to 2; varying respectively as follows: for "V" threads, oldo to .055 inch; for U. S. S. and Whitworth threads, .005 to .027 inch. These are found by adding to the size at bottom of thread, $\frac{1}{4}$ of the pitch for "V" threads, and $\frac{1}{6}$ of the pitch for U. S. S. and Whitworth, the pitch being equal to 1 inch divided by the number of threads per inch. In practice it is better to use a larger drill if the exact size called for cannot be had.

Sîze	of	S	ize of Dri	11	Size		Size of Drills		
Tap	No. of Threads	U. S. S.	V	W	Tap	Thre	U. S. S.	V	w
3 (1) 4 (6) (3) (8) (7) (1) (2) (2) (3) (3) (4) (3) (4) (3) (7) (8) (1) (1) (1) (1) (1) (1) (1) (1) (1) (1	24 20 18 16 14 13 12 12 11 11 10 10	.138 .191 .248 .302 .354 .409 .405 .518 .581 .632 .695	.111 .184 .239 .293 .345 .399 .391 .453 .506 .568 .680	.129 .192 .249 .303 .355 .410 .403 .466 .520 .583 .697 .747	156 I I 16 I I 188 I I 188 I I I 188 I I I I I I I	9 8 8 7 7 6 6 5 5 5 5 5 5 4 1 2 4 1 2	.808 .854 .917 .957 1.082 1.179 1.304 1.412 1.390 1.515 1.640 1.614 1.739	.790 .832 .894 .932 I.057 I.144 I.269 I.372 I.347 I.472 I.597 I.566 I.691	.810 .856 .919 .960 1.085 1.182 1.307 1.416 1.394 1.519 1.644 1.619

A very simple rule, which is good enough in many cases, is: Subtract the pitch of one thread from the diameter of the tap.

A $\frac{3}{8}$ -inch tap 16-thread would be $\frac{3}{8}$ minus $\frac{1}{16} = \frac{5}{16}$ drill; a $\frac{3}{4}$ -inch tap, ten-thread, would be $\frac{3}{4}$ minus $\frac{1}{10} = \frac{75}{100}$ or 0.75 — 0.10 = $\frac{65}{100}$ or 0.65, or a little over $\frac{5}{8}$ of an inch, so a $\frac{3}{8}$ -inch drill will do nicely. With a 1-inch tap we have $1 - \frac{1}{8} = \frac{7}{8}$ -inch drill, which is a little large but leaves enough thread for most cases.

TAP DRILLS FOR S. A. E. (A. L. A. M.) THREADS

			· · · · · · · · · · · · · · · · · · ·		
Size of Tap	Threads per Inch	Size of Drill, Inches	Size of Tap	Threads per Inch	Size of Drill, Inches
14 6 7 8 3 8 7 7 8 1 2 9 8 8 1 1 0	28 24 24 20 20 20 18 18 16	7 2774-14788 (7-47-14788 7-147-14788 7-147-14788 7-147-147-147-147-147-147-147-147-147-14	2 47:18 I 1:18 I 1:44 - 0:28-13:1	16 14 14 12 12 12 12	4 3 6 4 5 6 6 4 1 6 1 6

The tap should be between 0.002 and 0.003 inch large for clearance between top and bottom of threads.

TAP DRILLS

For Machine Screw Taps

These drills will give a thread full enough for all practical purposes but not a full thread as this is very seldom required in practical work, Further data along this line will be found in the tables which follow.

TAP DRILLS

Sizes of Taps	No. of Threads	Sizes of Drills	Sizes of Taps	No. of Threads	Sizes of Drills
2 2 2 2	48 56 64 40 48	48 46 45 48	12 13 13	24 20 24 20	19 17 15
. 3 3 3 4 4	48 56 32 36	47 45 45 43	14 14 15	22 24 18 20	13 11 12 10
4 4 5 5 5 5 6 6	40 30 32 36	42 41 40 38	15 16 16 16	24 16 18 20	7 10 7 5
6	40 30 32 36	36 39 37 35	16 17 17 17	24 16 18 20	1 7 4 2
6	40 28 30 32	33 32 31 30	18 18 18	16 18 20 16	B C
7 7 7 8 8 8	24 30 32 24	31 30 29 29	19 19 20 20	18 20 16 18	D E E E
9 9 9 9	28 30 32 24	27 26 24 26	20 22 22 22	20 16 18	F H I K
10 10	28 30 32 24	24 23 21	24 24 24 26 26	16 18 14 16	L M O P
11 11 12 12	28 30 20 22	19 18 21	28 28 28 30 30	14 16 14 16	R S T U

 $\qquad \qquad \text{Dimensions for Twist Drills}$ for boring holes to be threaded with u. s. form of thread

FOR BORING HOLES TO BE THREADED WITH U. S. FORM OF THREAD TAPS $\frac{1}{16}$ to $\frac{67}{4}$ INCH DIAMETER

Diam- eter Inches	No. of Threads to the Inch	Exact Diameter Bottom of Thread Inches	Gage No. of Drill	Diam- eter Inches	No. of Threads to the Inch	Exact Diameter Bottom of Thread Inches	Gage No. of Drill
1 0 1 0 2 3 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	60	.041	57	$\frac{1}{4}$	26	.200	6
$\frac{1}{1.6}$	64	.042	56	64	56	.055	53
32	48	.067	50	5 6 4	60	.056	53
$\frac{3}{32}$	50	.068	50	64	40	.077	46
$\frac{3}{32}$	56 60	.071	49	7 6 4	44	.080	45
$\frac{3}{3}$ 2	60	.072	48	64	48	.082	44
18	40	.093	41	64	32	.100	38
$\frac{1}{8}$	44	.096	40	64	36	.105	38 36
1 8	48	.098	39	64	40	.108	34
$\frac{5}{32}$	32	.116	31	$\frac{1}{6}\frac{1}{4}$	32	.131	29
$\frac{5}{3}$ 2	36	.120	31	$\frac{1}{6}\frac{1}{4}$	36	.136	28
$\frac{5}{32}$	40	.124	30	$\frac{1}{6}\frac{1}{4}$	40	.139	28
$\frac{3}{1.6}$	24	.133	29	$\frac{13}{64}$	24	.149	24
$\frac{3}{16}$	28	.141	27	134	28	.157	21
$\frac{3}{16}$	30	.144	26	13	32	.162	19
$\frac{3}{16}$	32	.147	25	$\frac{1}{6}\frac{3}{4}$	36	.167	18
3 1 6	36	.152	23	15	24 28	.180	13
$\frac{7}{32}$	24 28	.164	19	6 4	28	.188	10
$\frac{7}{32}$.172	16	15 6 4	32	.194	8
$\frac{7}{3}$ 2	32	.178	14	$\frac{15}{64}$	36	.198	7
$\frac{7}{32}$	36	.183	12	64	18	.193	9 5 3 2
14	18	.178	14	$\frac{17}{64}$	20	.201	5
$\frac{1}{4}$	20	.185	12	$\frac{1}{6}\frac{7}{4}$	24	.211	3
$\frac{1}{4}$	22	.190	10	. \$4.5 \$6.00 \$6.7 \$6.7 \$6.7 \$6.7 \$6.7 \$6.7 \$6.7 \$6.7	26	.216	
14	24	.196	8	$\frac{1}{6}\frac{7}{4}$	32	.225	1

DRILLS AND REAMERS FOR DOWEL PINS

Sizes of Ro	D	DRILLS AND I	REAMERS FOR	Drive Fits	DRILLS FOR CLEARANCE		
No. of Gage (Stubbs Steel Wire)		Size of Drill	Dia. of Drill	Dia. of Reamer	Size of Drill	Dia. of Drill	
54 45 33 30 21 10	.055 .081 .112 .127 .157 .191 .252 .315 .377 .439 .503 .628	" 47 " 36 " 31 " 24 " 13 C \$\frac{1}{5}\$ Reamer Drill	.052 .0785 .1065 .120 .152 .185 .242 .307 .366 .427 .489 .616	.110 .125 .155 .189 .2502505 .3125313 .3753755 .4375438 .5005005 .6256255 .7507505	No. 54 " 46 " 33 " 39 " 22 " 11 F O	.055 .081 .113 .1285 .157 .191 .257 .316	

DOUBLE DEPTH OF THREADS

Threads per in.	V Threads D D	U.S. St'd D D	Whit. St'd D D	Threads per in.	V Threads D D	U.S. St'd D D	Whit. St'd D D
						.	
2	.86650	.64950	.64000	28	.06185	.04639	.04571
$2\frac{1}{4}$.77022	•57733	.56888	30	.05773	.04330	.04266
$2\frac{\hat{a}}{8}$.72960	.54694	.53894	32	.05412	.04059	.04000
2 2 3 4 7 8 2 2 4 7 8	.69320	.51960	.51200	34	.05097	.03820	.03764
$2\frac{5}{8}$.66015	.49485	.48761	36	.04811	.03608	.03555
$2\frac{3}{4}$.63019	.47236	46545	38	.04560	.03418	.03368
$2\frac{7}{8}$.60278	.45182	.44521	40	.04330	.03247	.03200
3	•57733	.43300	.42666	42	.04126	.03093	.03047
$3\frac{1}{4}$ $3\frac{1}{2}$	•53323	.39966	.39384	44	.03936	.02952	.03136
$3\frac{1}{2}$.49485	.37114	.36571	46	.03767	.02823	.02782
4	.43300	.32475	.32000	48	.03608	.02706	.02666
$4\frac{1}{2}$.38488	.28869	.28444	50	.03464	.02598	.02560
5_	.34660	.25980	.25600	52	.03332	.02498	.02461
$\frac{5\frac{1}{2}}{6}$.31490	.23618	.23272	54	.03209	.02405	.02370
	.28866	.21650	.21333	56	.03093	.02319	02285
7 8	.24742	.18557	.18285	58	.02987	.02239	.02206
8	.21650	.16237	.16000	60	.02887	.02165	.02133
9	.19244	.14433	.14222	62	.02795	.02095	.02064
10	.17320	.12990	.12800	64	.02706	.02029	.02000
11	.15745	.11809	.11636	66	.02625	.01968	.01939
$11\frac{1}{2}$.15069	.11295	.11121	68	.02548	.01910	.01882
I 2	.14433	.10825	.10666	70	.02475	.01855	.01828
13	.13323	.09992	.09846	72	.02407	.01804	.01782
14	.12357	.09278	.09142	74	.02341	.01752	.01729
15	.11555	.08660	.08533	76	.02280	.01714	.01673
16	.10825	.08118	.08000	78	.02221	.01665	.01641
18	.09622	.07216	.07111	80	.02166	.01623	.01600
20	.08660	.06495	.06400	82	.02113	.01584	.01560
22	.07872	.05904	.05818	84	.02063	.01546	.01523
24	.07216	.05412	.05333	86	.02015	.01510	.01476
26	.0661	.04996	.04923	88	.01957	.01476	.01454
27	.06418	.04811	.04740	90	.01925	.01443	.01422
	·		·				

This gives the depth to allow for a full thread in a nut or similar piece of work for threads for 2 to 90 per inch, regardless of the diameter. A special nut for a 2-inch bolt, 20 threads per inch, U. S. Standard would have a hole 2. — .06495 = 1.93505 inches in diameter bored in it.

SIZES OF TAP DRILLS FOR TAPS WITH "V" THREAD

$\begin{array}{cccccccccccccccccccccccccccccccccccc$	\$\tag{2}\tag{3}\tag{3}\tag{5}\	Diam. Tap, in Inches
	52 54 56 60 32 36 40 48 30 40 32 36 40 32 36 40 32 36 40 32 36 40 32 36 40 30 40 40 30 40 40 40 40 40 40 40 40 40 40 40 40 40	Threads per Inch
	41 43 44 43 42 41 39 41	Size of Drill, Number
24 No. 20 28 No. 17 30 No. 16 32 No. 15 24 No. 15 25 No. 15 26 No. 12 26 No. 12 37 No. 16 27 No. 16 28 No. 17 30 No. 16 29 No. 12 31 No. 16 20 No. 12 32 No. 10 32 No. 10 33 No. 10 35 No. 10 36 No. 10 36 No. 10 37 No. 10 38 No. 10 39 No. 10 18 No. 17 20 No. 14 24 No. 9 16 No. 10 18 No. 10 19 No. 10 18 No. 10 18 No. 10 19 No. 10 18 No. 10 18 No. 10 19 No. 10 19 No. 10 18 No. 10 19 No.	[2] [2] [2] [2] [2] [2] [2] [2] [2] [2]	Diam. Tap, in Inches
No. 20 No. 17 No. 16 No. 15 No. 15 No. 16 No. 17 No. 16 No. 17 No. 17 No. 16 11 12 13 13 13 13 13 13 13 13 13 13 13 13 13	14	Threads per Inch
1	No. 12 No. 10 No. 10 No. 14 No. 9 No. 10 No. 3 No. 3 No. 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	Size of Drill
12	I	Diam. Tap, in Inches
1 1 2 2 2 2 2 2 2 2	14 10 11 12 10 11 12 11 12 10 11 12 10 11 12 10 11	Threads per Inch
$\begin{array}{c} \mathbf{I}_{\frac{3}{3}\frac{7}{2}} \\ \mathbf{I}_{\frac{3}\frac{7}{2}} \\ \mathbf{I}_{$		Size of Drill, Inches
	7.57.13 4 0.51.51.45 45 50.51.51.51 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	Diam. Tap, in Inches
77 77 77 77 77 77 66 66 66 66 66	$\begin{array}{cccccccccccccccccccccccccccccccccccc$	Threads per Inch
I I I I I I I I I I I I I I I I I I I	I I I I I I I I I I I I I I I I I I I	Size of Drill, Inches

This table gives similar information but in a way that would be more convenient in some cases.

TAPPING AND THREADING SPEEDS

For tapping in cast iron, the F. E. Wells & Son Company, Greenfield, Mass., recommends the following speeds:

$\frac{1}{4}$	<u>3</u>	$\frac{1}{2}$	<u>5</u> 8	$\frac{3}{4}$ inch holes
382	255	191	153	127

using an oil or soda compound.

For soft steel and iron:

1/4	<u>3</u> 8	$\frac{1}{2}$	<u>5</u>	$\frac{3}{4}$ inch holes
229	153	115	91	76

using oil as a lubricant.

The National Machine Company, Hartford, Conn., uses 233 revolutions per minute up to $\frac{1}{4}$ inch diameter and 140 revolutions per minute for sizes between $\frac{1}{4}$ and $\frac{1}{2}$ inch, using a screw-cutting oil as a lubricant.

They tap holes as deep as four tap diameters by power.

For threading cast iron in machines of the bolt-cutter type, the Landis Machine Company, Waynesboro, Penn., gives these speeds:

1/4	38	$\frac{1}{2}$	3 4	I	$I^{\frac{1}{2}}$	2 inches
200	150	125	100	85	55	45

with petroleum as a lubricant.

For soft steel and iron:

1/4	38	$\frac{1}{2}$	34	1	$I^{\frac{1}{2}}$	2 inches
280	220	175	140	115	75	60

with compound or screw-cutting oil, using a $2\frac{1}{2}$ -inch belt at 1200 feet per minute. The speeds are for high-speed steel dies. Some users run the machines at a much higher rate.

THREADING PIPE

The Bignall & Keeler Manufacturing Company, Edwardsville, Ill., aims to have its pipe-threading machines run at a cutting speed of 15 feet per minute. The machine for handling pipe from $\frac{1}{4}$ to 2 inches uses a $3\frac{1}{2}$ -inch belt at about 940 feet per minute. They advise nothing but lard oil on the dies.

The Standard Engineering Company, Ellwood City, Penn., also recommends a cutting speed of 15 feet per minute, using a 3-inch

belt at 730 feet per minute.

It will be understood in all the cases cited that the figures given are merely a guide as to what can be done and not record performances in any particular. Soft stock can be run very fast, and hard, gritty stock is very hard on the dies.

The only general rule, in the case of dies, is to run as fast as possible

without undue heating of the dies.

DRILL END LENGTHS

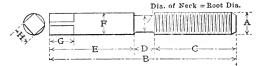
It is often necessary in designing brass castings to allow for drilling to a certain depth so as to give the thickness of metal A necessary at the bottom of the hole to withstand pressure.

at the bottom of the hole to withstand pressure.

The table gives the dimension C for usual sizes of drills. This is deducted from B to give the actual thickness of metal at A.

DRILL END LENGTHS

				DRILL	END LE	NGTE	15				
Dia.	С	C in near- est 1/64	Gage No.	Dia.	С	C in near- est 4	Gage No.	Dia.	С	C in near-est 1	
CONSTRUCTION OF THE PROPERTY O	0.60086 0.58208 0.56331 0.51453 0.52576 0.50698 0.48520 0.46942 0.45905 0.43187 0.337554 0.337554 0.337754 0.337754 0.337754 0.235776 0.23775 0.23775 0.23775 0.23775 0.23775 0.23141 0.22532 0.24410 0.23471 0.22532 0.105033 0.20655 0.111266 0.15022 0.11083 0.13144 0.12053 0.11266 0.11226 0.11266 0.11226 0.11266 0.11267	@#####################################	1 2 3 4 4 5 5 6 7 8 8 9 10 11 12 13 14 15 15 11 17 18 19 2c 2c 23 3c 24 25 26 27 28 3c 33 3c 35 3c 37 38 36 37 38 39 40	0.2280 0.2210 0.2130 0.2020 0.2020 0.2055 0.2040 0.2055 0.2040 0.1950 0.1950 0.1950 0.1850 0.1850 0.1870 0.1770 0.1730 0.1670 0.1570 0.1540 0.1520 0.1495 0.1495 0.1440 0.14285 0.1360 0.1160 0.1065 0.1040 0.1065	0.06850 0.06640 0.06140 0.06140 0.06129 0.06174 0.06129 0.053888 0.05378 0.05578 0.05578 0.05578 0.05518 0.05197 0.0107 0.0107 0.01401 0.01360 0.01211 0.01401 0.01360 0.033861 0.033861 0.03385 0.03385 0.03385 0.03395 0.03395 0.03395		41 42 43 44 44 45 46 47 48 49 50 51 52 53 54 55 56 66 66 66 67 70 70 70 70 70 80 80 80 80 80 80 80 80 80 8	0.0960 0.0935 0.0890 0.0860 0.0820 0.0730 0.0730 0.0730 0.0550 0.0550 0.0550 0.0455 0.0410 0.0400 0.0380 0.0370 0.0380 0.0330 0.0380 0.0320 0.0380 0.0320 0.0380 0.0320 0.0380 0.0320 0.0380 0.0380 0.0370 0.0380 0.0370 0.0380 0.0370 0.0380 0.0370 0.0380	0.02884 0.02809 0.02674 0.02584 0.02358 0.02358 0.02103 0.02103 0.02103 0.02103 0.02103 0.011652 0.01562 0.01262 0.01262 0.01262 0.01262 0.01262 0.01172 0.01172 0.01172 0.01172 0.01052 0.00681 0.00681 0.00681 0.00675 0.00676 0.00676		
$\frac{3}{3^{\frac{1}{2}}}$ $\frac{1}{16^{\frac{1}{6}}}$ $\frac{1}{32}$	0.02817 0.01878 0.00939	32 32 		- 6-							

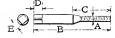


DIMENSIONS OF MACHINE SCREW TAPS

Number of Tap	Diameter of Tap	Number of Threads per Inch	Total Length	Length of Thread	Length of Neck	Length of Shank	Diameter of Shank	Length of Square	Size of Square	No. of Flutes
	A		В	С	D	Е	F	G	н	
1 1½ 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16 18 20 22 24 26 28 30	.071 .081 .089 .101 .113 .125 .141 .154 .206 .234 .246 .261 .272 .298 .325 .378 .404 .430 .456	64 56 56 48 36 32 32 32 30 24 24 22 20 20 18 16 16 16 16	I I I I I I 2 2 2 2 2 2 2 2 2 2 2 2 2 2	9 16 9 16 9 16 9 16 11 24 34 33 4 33 4 31 7 18 7 18 7 18 1 1 1 1 1 1 1 1 1 1 1 1	House the second of the second	$\begin{array}{c} 1_{1}^{3}_{1}^{5}_{1}^{5}_{1}\\ 1_{1}^{5}_{1}^{5}_{1}\\ 1_{1}^{5}_{1}^{5}_{1}\\ 1_{1}^{5}_{1}^{5}_{1}\\ 1_{1}^{5}_{1}^{5}_{1}\\ 1_{1}^{5}_{1}^{5}_{1}\\ 1_{1}^{5}_{1}^{5}_{1}\\ 1_{1}^{5}_{1}^{5}_{1}^{5}_{1}^{5}_{1}\\ 1_{1}^{5}_{1}^{5}_{1}^{5}_{1}\\ 1_{1}^{5}_{1}^{5}_{1}^{5}_{1}\\ 1_$.125 .125 .125 .125 .125 .141 .154 .166 .180 .221 .224 .246 .261 .272 .298 .325 .404 .430 .456	9 16 8 16 9 16 9 16 7 13 73 73 73 75 7 15 7 15 14 14 14 14 15 15 15 15 16 16 16 16 16 16 16 16 16 16 16 16 16	\$\frac{1}{15}\$\cdot \frac{1}{15}\$\cdot \frac{1}{15}	3 3 3 3 3 3 3 3 3 4 4 4 4 4 4 4 4 4 4 4

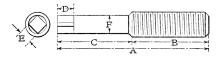
These are for the American Screw Company's Standard screws that have been in use for many years, $\,$

Machine Screw Taps—(Wells Bros. Co.)



Size	Threads	Size		tside Thread	Whole	Length	Length	Diam.
Size	per Inch	Steel	Max.	Min.	Length	Thread	Square	Square
				A	В	С	D	Е
-	80	.147	.063	.061	1 1 5 1 3 2 2 1 2 3 2 1 1 6 5 1 1 1 6 5 1 1 1 6 5 1 1 1 1 6 5 1 1 1 1	9 Tā	7.7	.112
ī	7.2	.147	.076	.074	I 5	5 8	379	.112
2	64	.147	.oSq	.088	1 2 3	31	37.	.112
3	56	.147	.103	.101	137	9 George 183 el 4 552 el 6 e 6 7 le 60 153 6 163	32	.112
4	48	.147	.116	.114	115	2 5 3 2	32	.112
4 5 6	44	.147	.120	.127	131	13	32	.112
ŏ	40	.147	.142	.140	2 3 2	13	3/2	.112
7	36	.165	.155	.153	2 3 2	7 8	1	.124
7 8	36	.174	.169	.167	232	29	Į į	.131
9	32	.187	.181	.179	215	15	1 1	.140
10	30	.200	.194	.192	2 1 13 1 2 9 16 7 2 9 16 7 2 9 16 7 2 9 16 7 2 9 16 7 2 9 16 16 16 16 16 16 16 16 16 16 16 16 16	31		.150
II		.215	.206	.204	2 1	I	1 1	.161
I 2	28	.226	.220	.218	218	132	32	.170
13		.252	.239	.237	2 17	I 3 2	32	.189
1.4	24	.252	.246	.244	2 1 9 2	1 1	3 2	.189
15		.258	.258	.256	2 5/8	I 1/8	32	.194
16	22	.278	.273	.270	231	I 1/8	33	.200
18	20	.304	.299	.296	2 3/4	I 1 8 I 1 8 I 3 3	1 ⁵ 6	.228
20	20	.332 .32S	.325	.322	2 1 3 1 3	116	15	.249
22	18	.370 .366	.351	.348	2 7 8	1 3 3	1 ⁵ 6	.278
2.1	16	.387 .383	-377	-374	215	1 32	1 5€	.215
26	16	.418 .414	.403	.400	3	1 ₁₈	11 32	.236
28	1.1	·449 ·445	.430	.427	3 3 5	I 3	11 32	.249
30	14	.480	.456	-453	3 1/4	1 7 d	11 32	.268

	Pitch	Dia.	0.	Pitch	Dia:	c:	Pitch	Dia.
Size	Max.	Mın.	Size	Max.	Min.	Size	Max.	Min.
0 × 80 1 × 72 1 × 64 2 × 56 3 × 48 4 × 40 4 × 36 5 × 44 5 × 40 6 × 36	.0538 .066 .065 .078 .0767 .0897 .0579 .1010 .0984 .0967 .1129 .1115 .1098	.0528 .065 .064 .0769 .0756 .0886 .0868 .0998 .0972 .0955 .1116 .1108 .1085 .1232	7 × 36 7 × 32 7 × 30 8 × 36 8 × 32 8 × 30 9 × 32 9 × 30 10 × 30 10 × 32 10 × 24 11 × 24	.1359 .1337 .1325 .1489 .1467 .1454 .1584 .1584 .1582 .1716 .1729 .166 .1961	.1345 .1323 .131 .1475 .1453 .1569 .1517 .170 .1713 .1647 .1944 .1997	16 × 22 16 × 20 18 × 20 18 × 18 20 × 20 20 × 18 22 × 18 22 × 16 24 × 16 26 × 16 26 × 16 28 × 14 30 × 16	.2421 .2594 .2652 .2618 .2912 .2878 .3138 .3094 .3354 .3614 .3557 .3874 .3818 .4154	.2403 .2374 .2598 .2804 .2858 .3118 .3074 .3378 .3334 .3594 .3537 .3854 .3797 .4134
5 × 36 6 × 40	.1098 .1246	.1085	12 × 28 12 × 24	.1961	.1944 .1907	28 × 16 28 × 14	.38	74 18 54

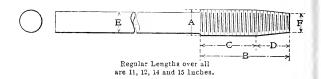


DIMENSIONS OF HAND TAPS (WELLS BROS. Co.)

Small Shank Taps U. S. S., Whit. and all V Taps Incl.

						1	
Size	Pitch	Total Length	Length Thread	Length Shank	Length Square	Diam. Square	Diam. Shank
		A	В	С	D	Е	F
140 (1915) (1915) (1916	20 18 14 St'd 11 10 9 8 St'd 6 6 6 6 St'd 4 4 5 5'd 4 4 5 5'd 4 4 5 5'd 4 5 5'd 4 5 5 5 6 6 6 6 6 6 6 6 6 6 6 6 6	1916 a triangle of the state of	다입니다. 너무 네이트 아이트 네트	다음 다음 전체에 보다 이 마음	02000000000000000000000000000000000000	.134 .176 .217 .286 .321 .359 .406 .523 .569 .600 .617 .719 .766 .813 .831 .878 .925 .971 .978 .1025 .1.072 .1.139 .1.39 .1.38 .1.327 .1.421 .1.575 .1.688 .1.768 .1.768	.173175 .228230 .283285 .330332 .373375 .479 .542 .590 .652 .607 .759 .800 .862 .896 .959 1.021 1.084 1.108 1.170 1.233 1.296 1.305 1.430 1.430 1.430 1.510 1.582 1.644 1.769 1.894 2.010 2.100 2.225 2.350 2.475 2.542

72 TAPS



DIMENSIONS OF TAPPER TAPS

Diameter of Tap		nber ireads Inch	Length of Thread	Length of Straight Part	Length of Chamfered Part	Diam Shar	eter of nk E		eter of at F	No. of Flutes
A	U.S. St'd	V. St'd	В	С	D	U S. St'd	V. St'd	U.S. St'd	V. St'd	
145 [C 255] [155] [155] [25] [20 18 16 14 13 12 11 10 10 9 9 8 7 7 6 6	20 18 16 14 12 12 11 10 10 9 9 8 7 7 6 6	1 2 2 2 2 2 2 2 2 2 3 3 3 3 3 3 3 3 4 4	1 S 4 4 S S 6 E 5 S 5 4 4 5 S 5 5 5 5 4 4 5 5 5 5 5 5 5	5834347878 I I I I I I I I I I I I I I I I I I I	0.225 0.280 0.330 0.335 0.440 0.490 0.555 0.605 0.670 0.720 0.780 0.820 0.925 1.050	0.250 0.300 0.340 0.455 0.515 0.560 0.625 0.675 0.770 0.860 0.985	0.234 0.287 0.338 0.393 0.446 0.561 0.673 0.722 0.783 0.828 0.928 1.053 1.147	0.210 0.261 0.307 0.348 0.411 0.462 0.523 0.570 0.631 0.675 0.736 0.775 0.869 1.075	4 4 4 4 4 4 4 4 4 4 4 4 4 4

Note. — Tapper taps differ from machine taps in not having a square on the end of the shank. They are used in nut tapping machines, the nuts being run over the tap on to the shank and when full the tap is removed and the nuts slid off. The tap is then replaced for another lot of nuts.

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DIMENSIONS OF BRIGGS STANDARD PIPE TAPS

(Wells Bros. Co.)

11,73		_	Length				-	Diameter			•		Straight	ght
per Inch	V q	В	ပ	А	ы	Œ	G	E		и	spųL	Flute	Outside	Pitch
27	- N	1	m)sc	m)so	E 25	.373	.309	.420	ండ్ల	.234	27	4	.402	.374
18	21.		H H	Ŋ,	ارع د	.493	.397	.559	16	.328	18	4	.536	.494
18	21.8		T S	-400	, 3 G	.628	.532	.694	10	.421	18	4	029.	.628
1.4	, c.		H 6/4	1000	P-21	.779	959.	.865	11	.515	Į.	4	.834	.780
14	3 5	I S	H 814	15.	F.0	.989	998.	1.075	5.71 5.71	629.	14	4	I.043	686.
II	ω 614	H 3	8	E 2	-14	1.240	1.001	1.350	1.00 × 1.00	.843	11 2	Ŋ	1.305	1.240
II	4	H	2	500	~	1.584	1.434	1.693	IIg	.984	II 2	25	1.648	1.583
II	4	E H	C1 -402	ı	-44	1.822	1.672	1.932	I 3	1.125	112	9	1.888	1.823
II	4	H	61	I 8	~*	2.297	2.147	2.405	H 1-100	1.406	11 2	9	2.361	2.296
∞	7.7 46	21.0	216	I 4	m/so	2.761	2.545	2.921	2 4	1.687	∞	9	2.850	2.756
×	9	6,00	ς3 κ:α	E)30	m)x0	3.383	3.167	3.547	() ()	1.968	8	9	3.472	3.378
×		211	313	1 3	es 20	3.879	3.663	4.047	213	2.150	8	9	3.969	3.875
8	6 3	2	4	K)X	en(00	4.375	4.159	4.547	3	2.250	00	9	4.465	4.371

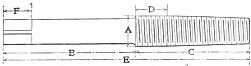
Taper 3" per foot = 1_a " per inch. "F" = Diameter at point before plugging.

STOVE BOLT TAPS (WELLS BROS. Co.)

rned	山	
Not Turned	÷	A .
	p	

		B C D	O	O 1 1 1 1 2 2 2 2 2 2	O 1 2 2 2 2 1 1 1 1 1 1 1 1 1 1 1 1 1 1	O 1 1 1 1 1 1 1 2 1 1 2 2 2 2 2 2 2 2 2	O 13 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	C D I S S S S S S S S S S S S S S S S S S
_	a 		<u> </u>	전 이외 하고 1년의	다. 이용 이용 호수 1년의 변화	어'::: (이 어어 어어 H	어'에 1년61 대연3 대연 어'에 1년61 대연3 대연5	1 000 000 11.000
				colto	പ്രേ ത്ര	ල්ල හැන <u>H</u>	ටෑ න නේන නේන	
ද් <u>ස්වූ</u>	I 3 2 2 1		2, 12, 12, 12, 13, 14, 14, 14, 14, 14, 14, 14, 14, 14, 14			232 232 2 1 I	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	ର୍ଷ ମଧ୍ୟ ମଧ୍ୟ ହାଉ ଆଷ ମଧ୍ୟ ମଧ୍ୟ ହାଉ
.0063 20 U.S.S. .0069 18 U.S.S.	20 U.S.S. 20 U.S.S. 20 U.S.S. 18 U.S.S.	.0069 18 U.S.S.	000	16 U.S.S.	9800.	.0.0.0	.0114 .0114 11 U.S.S.	.0114 .0114 .11 U.S.S.
			021.	.150	091.		.194	
Min.		811.	.150	.180	.204		.231	.231
-	Max.	.120	.152	.182	.206	_	.233	.233
	Min.	.125	991.	.200	.227	_	.260	.317
	Max.	.127	891.	.202	.229		.263	.320
Size	Used	.147	.174	.215	.252	,	.278	.328
No. Th'ds		2	- 82	24	22		18	81
	Inch	32				_		



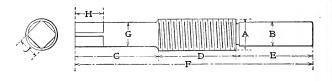


Diam. of Shank Root Diam, less 0.015"

DIMENSIONS OF TAPER DIE TAPS

Diameter of Tap	Length of Shank	Length of Thread	Length of Straight Thread	Total Length	Length of Square	Size of Square	Number of Flutes
Α	В .	С	D	Е	F	G	
145 T 385 T 15 25 T 15 25 T 15 25 T 15 24 25 T 15 16 T 15 14 25 T 15 15 25 25 25 25 25 25 25 25 25 25 25 25 25	1 1 1 1 1 2 2 2 2 3 3 3 3 3 3 3 3 3 3 3	2 2 3 3 3 3 3 4 4 4 4 5 5 5 5 6 6 6 6 7 7 7	145,05857,0557,0551,0551,0543,057,550 I 181,4436,1615634448 I I I I I I I I I I I I I I I I I I I	31 4 12 5 12 5 6 12 7 12 8 8 8 8 9 14 12 9 9 14 10 10 10 10 11 10 10 11	9 1 5 8 1 1 6 1 6 4 4 3 6 6 3 6 7 5 7 8 7 8 7 5 5 6 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	1865 386 1 146 887 5 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	5 5 5 5 5 5 5 6 6 6 6 6 6 6 6 7 7 7 7 8 8 8

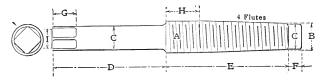
76 TAPS



DIMENSIONS OF SELLERS HOBS

Diameter of Hob	Number of Threads per Inch		Diameter of Pilot B		Length of Shank	Length of Thread	Length of Pilot	Total Length	Diameter of Shank G		Length of Square	Size of Square	No. of Flutes
A 	U.S. St'd	V. St'd	U.S. St'd	V. St'd	С	D	E	F	U.S.St'd	V. St'd	Н	J	
145/6387/6129/6381/1634/867/8516 1 1 1 1 1 1 1 1 1 2	20 18 16 14 13 12 11 10 10 9 9 8 7 7 6 6 6 5 1 ¹ / ₂ 5 5 5 1 ¹ / ₄ 1 ¹ / ₂ 1 ¹ / ₂ 1 ¹ / ₄ 1 ¹ / ₄ 1 ¹ / ₂ 1 ¹ / ₄ 1 ¹	20 18 16 14 12 11 10 10 9 9 8 7 7 6 6 5 5 1 4 1 2 4 4 4 4 4 4 4 4 4 4 4 4 4	$c_1^{(c)}c_2^{(c)}c_3^{(c)}-t_1^{(c)}c_3^{(c)}$	$\begin{array}{cccccccccccccccccccccccccccccccccccc$	2 2 16 16 16 16 16 17 17 17 17 17 17 17 17 17 17 17 17 17	$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	$ \begin{array}{c} 2\frac{1}{2} \\ 2\frac{3}{4} \\ 3 \\ 3\frac{1}{16} \\ 3\frac{1} \\ 3\frac{1}{16} \\ 3\frac{1}{16} \\ 3\frac{1}{16} \\ 3\frac{1}{16} \\ 3\frac{1}{16} \\ $	$\begin{array}{c} +\frac{1}{4}\frac{4}{4}\frac{1}{2} \\ +\frac{1}{4}\frac{1}{2} \\ -\frac{1}{4}\frac{1}{4} \\ -\frac{1}{4}\frac{1}{4}\frac{1}{4} \\ -\frac{1}{4}\frac{1}{4}\frac{1}{4} \\ -\frac{1}{4}\frac{1}{4}\frac{1}{4} \\ -\frac{1}{4}\frac{1}{4}\frac{1}{4}\frac{1}{4} \\ -\frac{1}{4}\frac{1}{4}\frac{1}{4}\frac{1}{4} \\ -\frac{1}{4}\frac{1}{4}\frac{1}{4}\frac{1}{4} \\ -\frac{1}{4}\frac{1}{4}\frac{1}{4}\frac{1}{4}\frac{1}{4} \\ -\frac{1}{4}\frac{1}{4}\frac{1}{4}\frac{1}{4}\frac{1}{4} \\ -\frac{1}{4}\frac{1}{4}\frac{1}{4}\frac{1}{4}\frac{1}{4}\frac{1}{4} \\ -\frac{1}{4}\frac{1}{4}\frac{1}{4}\frac{1}{4}\frac{1}{4}\frac{1}{4}\frac{1}{4}\frac{1}{4} \\ -\frac{1}{4}1$	0.170 0.225 0.280 0.330 0.385 0.440 0.490 0.555 0.605 0.715 0.780 0.825 1.055 1.270 1.375 1.475 1.600	0.150 0.200 0.250 0.300 0.340 0.455 0.515 0.560 0.675 0.770 0.860 0.985 1.070 1.200 1.265 1.390 1.475 1.600	34 34 34 34 36 36 36 36 78 78 78 78 55 55 55 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	13 13 16	6 6 6 6 8 8 8 8 8 8 10 10 10 10 10 11 12 12 12 12

Note. — The Sellers hob is designed to be run on centers, the work, such as hand or die chasers, being held against it and fed along by the lathe carriage.



STANDARD SQUARE-THREAD TAPS

Size	A	В	С	D	Е	F	G	Н	I
Diameter $\frac{5}{8}''$ (1)	514921 516921 508	$\frac{\frac{1}{2}}{\frac{1}{3}\frac{7}{27}\frac{7}{4}}$	$\frac{1}{2}$ $\frac{1}{2}$ $\frac{1}{2}$	35555 3555 38	3 ¹ / ₄ 3 ¹ / ₄	1/21/21/21	9 16 16 9 16	ocion ocion ocion	$\frac{7}{16}$ $\frac{7}{16}$ $\frac{7}{16}$ $\frac{7}{16}$
Pitch 8 (3)				35	3 \frac{1}{4} 3 \frac{1}{4}	$\frac{1}{2}$	$\frac{9}{16}$	5	16
Diameter $\frac{3}{4}$ "(1)	$\frac{41}{64}$ $\frac{45}{64}$ $\frac{3}{4}$	$\frac{37}{64}$ $\frac{5}{8}$ $\frac{11}{16}$	0)[00][00][0	3 4 3 4 3 4 3 4	3 8 3 8 3 8 3 8 3 8 8 8 8 8 8 8 8 8 8 8	1 1 2 1 2	3/4 3/4 3/4	ত ঝ হে ঝ হু ঝ	$\frac{7}{16}$ $\frac{7}{16}$ $\frac{7}{16}$ $\frac{7}{16}$
Pitch 6 (3)				34					
Diameter $\frac{7}{8}''$ (1)	$\frac{\frac{3}{4}}{\frac{1}{1}\frac{3}{6}}$	21/327 454 514	2 002 002 00 2 002 002 00	4	34 34 34 34	1 1 2 1 2 1	136 136 136 136	7 8 7 8 7 8	1 2 1 2 1 2
Pitch $4\frac{1}{2}$ (3)	7 8			4	$3\frac{3}{4}$	$\frac{1}{2}$		8	$\frac{1}{2}$
Diameter 1"(1)	$\frac{5}{6}\frac{7}{4}$	$\begin{array}{c c} \frac{51}{64} \\ \frac{7}{8} \\ \frac{15}{16} \end{array}$	56565656	$ \begin{array}{c} 4\frac{1}{2} \\ 4\frac{1}{2} \\ 4\frac{1}{2} \end{array} $	$4\frac{1}{4}$ $4\frac{1}{4}$	12 12 12 12	156 1656 1656 166	I	5 85 85 8
Lead D'BL $\frac{2}{5}''(3)$	I	$\frac{15}{16}$	51	$4\frac{1}{2}$	$4\frac{1}{4}$	$\frac{1}{2}$	$\frac{15}{16}$	I	<u>5</u> 8
Diameter $1\frac{1}{8}$ (1) (2) (3)	$ \begin{array}{c} \frac{61}{64} \\ 1\frac{1}{32} \\ 1\frac{5}{64} \end{array} $	55 64 15 16 16 16	5 05 05 05 4	$ 4\frac{3}{4} 4\frac{3}{4} 4\frac{3}{4} 4\frac{3}{4} 4\frac{3}{4} 4\frac{3}{4} $	48 48 48 48 48 48	$\frac{1}{2}$ $\frac{1}{2}$ $\frac{1}{2}$	I .	$1\frac{1}{8}$ $1\frac{1}{8}$ $1\frac{1}{8}$	11 16 11 16 11 16 11 16
Pitch $3\frac{1}{2}$ (4)	$1\frac{1}{8}$	116	55 64	$4\frac{3}{4}$	4 3/8	$\frac{1}{2}$	1	1 1/8 1 1/8	116
Diameter $1\frac{3}{8}''$.(1) (2) (3)	$1\frac{13}{64}$ $1\frac{9}{2}$ $1\frac{21}{64}$	$ \begin{array}{c c} \mathbf{I} & \frac{1}{8} \\ \mathbf{I} & \frac{3}{16} \\ \mathbf{I} & \frac{1}{6} & \frac{7}{4} \end{array} $	I \frac{1}{8} I \frac{1}{8} I \frac{1}{8}	5 5 5 5 5 5 5	$ \begin{array}{c c} 4\frac{5}{8} \\ 4\frac{5}{8} \\ 4\frac{5}{8} \\ 4\frac{5}{8} \end{array} $	න්තන්තන්තන්ත	1 \frac{1}{8} 1 \frac{1}{8} 1 \frac{1}{8}	1 38 1 38 1 38 1 38 1 38	787878
Lead D'BL ½"(4)	18	1 1 6	1 1/8	54	45	58	I 1/8	1 3/8	78

NOTE. — While in theory the thread and the space are both one half the pitch in practice it is necessary to make the thread a little more than half in order to allow clearance for the screw that goes into the threaded hole. The amount of this clearance depends on the character of the work and varies from .oor inch up. Some also make the tap so that the screw will only bear on the top or bottom and the sides.

FILES

FILES are designated both by the spacing of their teeth and the shape or cross-section of steel on which the teeth are cut; the size always referring to their length which is measured from the point cutting to the end of the file proper but the measurement never includes the tang which fits into the handle.

TERMS USED

The back of a file is the convex or rounding side of half-round, cabinet and other files having a similar shape.

A file is Bellied when it is full or large in the center.

A Blunt file is the same size its whole length instead of being tapered.

An Equalling file is one which looks blunt but which has a slight belly or curve from joint to tang.

A Float file is a coarse single cut made for use on soft metals or

wood and frequently used by plumbers.

A Safe-edge is an edge left smooth or blank so that the file will not

cut if it strikes against the side of a slot or similar work.

The Tang is the small pointed end forged down for fitting into the

handle.

Three square files are double cut and have teeth only on the sides, while taper saw files are usually single cut and have teeth on the edge as well as the sides. This makes the taper saw files broad on the edge or without sharp corners, while the three square files have very sharp corners.

A special angle tooth file is made for brass work. The first cut is square across the file, while the second is at quite an acute angle,

about 60 degrees from the first cut.

Doctor files are very similar to these except that the first cut is

about 15 degrees instead of being square across the file.

A lock file has safe edge and the teeth only go about one third the way across from each side leaving the center blank. The teeth are single cut.

HIGHT OF WORK

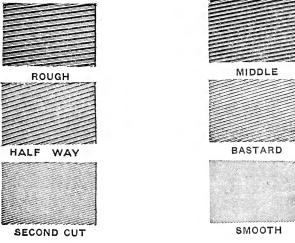
The work should be at a convenient hight which will usually vary from 40 to 44 inches for most men with an average of 42 inches. This means the hight of the work, not the bench.

Pickling Bath

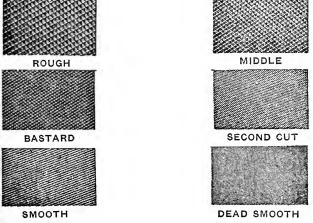
A good pickle to soften and loosen the scale on cast iron before filing is made of two or three parts of water to one part of sulphuric acid. Immerse castings for a short time.

For brass castings use a pickle of five parts water to one part nitric

acid.



Actual Tooth Spacing of Single Cut Files



Actual Tooth Spacing of Double Cut Files

80 FILES

THE TEETH OF FILES

The cut of a file or the number of teeth per inch vary with the length of the file itself and the kind of a file, and is a little confusing, as a rough cut in a small file may be as fine as a second cut of a larger size. The cuts used on regular 12-inch files are shown in the illustration and represents the practice of Henry Disston & Sons. The same makers also supply the table of cuts per inch used on their machines, which are as follows:

REGULAR TAPER FILES

Length, inches. — $2\frac{1}{2}$, 3, $3\frac{1}{2}$, 4, $4\frac{1}{2}$, 5, $5\frac{1}{2}$, 6, $6\frac{1}{2}$, 7, 8, 9, 10, Teeth per inch — 64, 56, 52, 50, 48, 46, 44, 42, 42, 40, 38, 36, 34. Slim Tapers — 64, 64, 60, 58, 56, 52, 50, 50, 46, 46, 44, 40, 38.

MILL FILE, BASTARD CUT

Length — 4, 5, 6, 7, 8, 9, 10, 11, 12, 13, 14, 15, 16, 17, 18, 20 in. Teeth — 56, 50, 48, 46, 44, 42, 40, 38, 36, 34, 32, 30, 28, 26, 24, 22 per inch.

FLAT FILE, BASTARD CUT

48, 42, 38, 36, 32, 30, 26, 24, 22, 20, 20, 18, 18, 16, 16, 14. Single cut files usually have teeth at about 25 degrees and in doublecut files the other cut is usually from 45 to 50 degrees. Fine machinists files are made in ten numbers from oo to 8.

THE SHAPES OF FILES

In the following pages the shapes of standard files are shown. The names are as follows:

 Metal saw — blunt. 6. Round or rat-tail. 12. Warding.

11. Crochet.

2. Three-square or tri- 7. Pippin. 13. Extra narrow 8 Knife. pillar. angular.

3. Barrette. 14. Narrow pillar. Crossing. 10. Half-round. 15 Pillar. 4. Slitting.

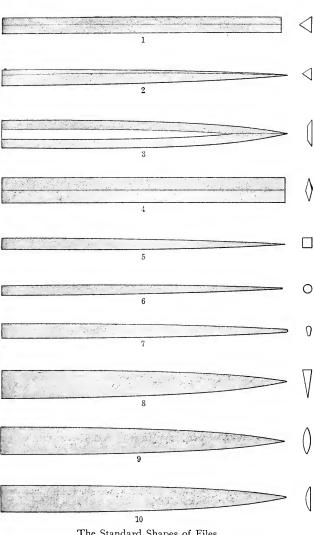
16 Hand.

5. Square. Files are cut in two ways, single and double The first has but a single line of cuts across the surface, at an angle with the file body but parallel to each other. The double-cut file has two lines of cuts, at an angle with each other, and the second cut being usually finer than the first. Some prefer the single cut, for filing in the lathe. Rasps have single teeth forced up with a punch.

The old method of designating the cuts were rough, coarse, bastard, second cut, smooth and dead smooth. Some makers are now using a series of numbers - usually eight to ten - instead of the six designations by name formerly employed. The uses of the various cuts depend on the shop in question and must be learned from

observation and experience in each case.

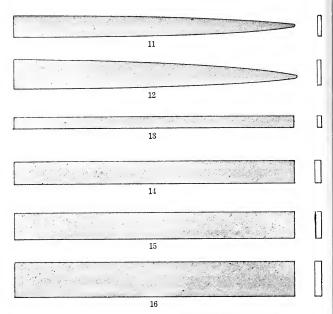
The grades of cut used by them run from No oo to No. 8, and while it is hard to exactly compare them with the old-style designations, it will be found that No. oo is about the same as a bastard, No. 1 as a second cut, No. 2 or 3 with a smooth, and Nos. 6 to 8 with a dead smooth file.



SHAPES OF FILES

81

The Standard Shapes of Files



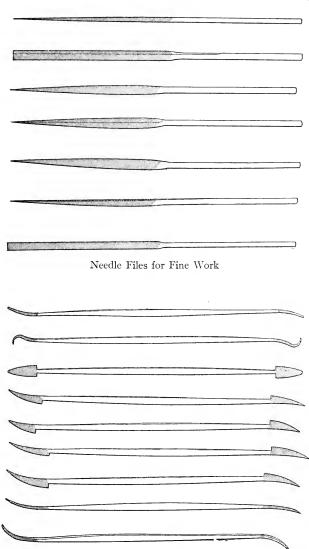
The Standard Shapes of Files

When a File Cuts Best

One who has given the matter careful attention, and has built file-testing machines, Edward G. Herbert, of Manchester, England, has come to the conclusion that a file does not cut best when it is new but after it has been used for some little time, say 2500 strokes or the filing away of one cubic inch of metal. Another curious feature is that its usefulness seems to come to a sudden instead of a gradual end.

A bastard file having 25 teeth to the inch, operating on a surface one inch square with a pressure of 30 pounds, which is about equal to heavy hand filing, gives 25 cutting edges about one inch long, which likens it somewhat to a broad cutting tool in a planer.

In cutting a file the metal is forced up in a sort of a bur, and occasionally the top of the tooth slopes over backward which is the reason that a file often cuts better after these are broken or worn off. Then, too, when a file is new all the teeth are not of the same hight and only a few points cut. As they wear down more teeth come into contact and do more work.



Die Sinkers Files or Riffles

WORK BENCHES

THE duties of a bench vary with the shop in which it is located according to the work that is to be done on it or at it. If it is simply a filing bench, the main requirement is that it support a vise firmly and at the proper hight. If an assembling bench, these are not the important features, and just what it does need depends on the kind

of work being handled.

For the average shop work we want a bench that is rigid; that will stand chipping and filing; that can be used in testing work on a surface plate or in handling jigs and fixtures; that will not splinter badly nor yet injure a tool should it happen to drop on it. For the toolmaker the cast-iron bench top has many advantages, but both the bench and the tool are very liable to be marred by dropping the tool on it, so that for general use we rely on wood as in the days of old, except that a bench with solid 2- or 3-inch planking the whole width is now too expensive to consider. We no longer want the bench braced up against the side of the shop but set it out from

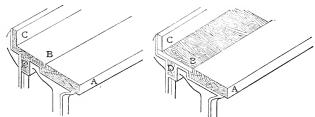


FIG. 1. Good for Ordinary Work.

FIG. 2. Another Method.

the wall to allow the heat to rise and the air to circulate, as well as giving the sprinklers a chance to get at a fire on the floor near the walls.

The use of a lighter board at the back has become so common that the New Britain Machine Company's design for a bench leg is made for this construction as shown in Fig. 1. This also shows the backboard B rabbeted to the plank A, which supports it all along the front edge, and it is also supported by the stringer D, which runs the whole length of the bench. These supports, in addition to the cross bearing of the legs every 6 or 8 feet, give the backboard a stiffness that was unknown where they are simply laid flush and not rabbeted and the stringer is absent.

Benches made without these supports are open to the serious objection that the backboard springs down when a heavy weight, such as a jig or surface plate, is put on the bench and throws them out of

level.

All cracks are more or less of a nuisance in bench work, but in this case any shrinkage can be taken up by wedging against the iron support of the board C and the edge of the backboard B.

Another style bench with this same leg is shown in Fig. 2. Here the front plank A and the backboard C are the same as before, but instead of having one backboard, this part of the bench is made up of narrow strips as B, fitting into rabbet in plank A and supported by the stringer D as before. These narrow boards can be either tongued and grooved hardwood flooring, or can be square edges, as preferred; in either case any shrinkage can be taken up by forcing the boards together.

A cheaper form of bench is shown in Fig. 3, where the heavy planking is entirely dispensed with and the boards B run the full width of the bench as shown. Running along the front, underneath the main boards is a soft plank A which supports the edge of the bench where the most work comes, and under the back is the 2 × 6inch stringer as before. Here, too, the boards can be either notched or square edge, each having its advocates; the objection raised against the tongue and groove being that the edges are apt to split off from heavy articles dropping on them. An advantage claimed for the boards running this way is that work going on or off the

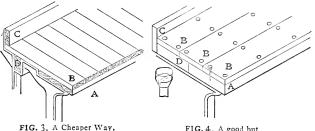


FIG. 4. A good but expensive Construction.

bench is always in the direction of the grain of the wood and that fewer splinters are formed on that account. In either Figs. 2 or 3 any local wear can be remedied by replacing the worn board with a Some object to the end of the grain at the front of a bench.

The material used in any of these can be varied to suit the individual requirements. Maple is generally considered the best wood for a bench, while others prefer ash. For the backboards hard pine is often used and even cheaper woods will answer if necessary, although it probably pays to use maple all through if you can afford it.

Still another style of bench is shown in Fig. 4 and one which was designed to be serviceable and have a long life without so much regard to first cost as the others. The bench leg was flat on top, the first layer of maple planks A and D and on top, narrower boards of the same material. These were fastened with long wood screws, holes being countered and plugged as shown.

The theory of this construction is that the boards are sure to be more thoroughly seasoned than the planks, consequently the planks will shrink the most and tend to draw the top boards closer together. It certainly makes a solid bench, but the first cost is rather high.

Benches are also occasionally built up from small blocks so as to present an end grain on the top, the same as butchers' blocks. One shop we know of surfaces these when worn by putting them on a Daniels' planer and substituting a circular saw for the swinging knives. This saws the top very smooth and leaves a good surface. Others glue up strips on edge and plane down to a smooth surface so as to do away with all cracks. Zinc or even heavy paper covers are often used where fine work is being assembled to prevent its finding its way into cracks and crevices.

The usual work bench is from 33 to 35 inches from the floor to the top, about 29 or 30 inches wide, and has the front plank 3 inches with backboards 1 inch thick. A cast-iron leg of this type weighs about

50 pounds.

SOLDERING

Almost every one thinks he can solder, yet if we examine the work carefully we will find that only about 10 per cent of the work is really done as it should be. Thorough soldering is frequently referred to as sweating, and it is remarkable the difference in strength between a well-fitted and "sweated" junction of the metals and

one as ordinarily soldered.

A point frequently overlooked is the important one of properly cleaning the surfaces to be joined. This is too often left for the flux to correct. Another neglected point is the selection of the flux to be used, although nearly all of the metals can be joined by the use of the same flux. The after effects resulting from improper cleaning after soldering are frequently worse than the good effects of the soldering. This is particularly noticeable in electrical work.

For strength, fit the parts accurately. The more accurate the fitting the stronger the result. Use a solder with as high a melting point as possible. Apply the proper heat as it should be. The nearer the temperature of work to be joined is brought to the fusing point of the solder the better will be the union, since the solder will

flow more readily.

Fluxes for Different Metals

There are on the market a number of fluxes or soldering salts that are giving good satisfaction. A form that is non-corrosive and very popular with electrical workers is the soldering stick in which the ingredients are molded into stick form about r inch diameter and 6 inches long.

The action and use of a flux in soldering are to remove and prevent the formation of an oxide during the operation of soldering, and to allow the solder to flow readily and to unite more firmly

with the surfaces to be joined.

For sheet tin, on the best work rosin or colophony is used; but owing to the ease of applying and rapidity of working, zinc chloride

or acid is more generally used. Beeswax can also be used, as also almost any of the pastes, fats or liquids prepared for the purpose.

For lead, a flux of oil and rosin in equal parts works very well. Tallow is also a good flux. Rosin or colophony is much used, and

zinc chloride will keep the surfaces in good condition.

Lead burning is a different operation from soldering, and at the present time almost a lost art. The surfaces must be bright and free from oxide; solder is not used as a flux, but a piece of lead and

rosin or oil.

For brass, zinc chloride or almost any of the soldering preparations is used. Care must be taken to remove any scale or oxide if a good joint is wanted. On new metal this is not much trouble, but on old or repair work it is sometimes exceedingly difficult. This is particularly noticeable on metal patterns that have been in use for some time. The scraper must be brought into use to remove it. Many use with considerable success an acid dip such as is commonly used by electro platers, for removing the oxide. Oily or greasy work can be cleaned by the use of potash or lye, but care must be exercised that the brass is not left too long in the solution, especially if it contains any joints previously soldered, since the action set up will dissolve the solder entirely or roughen up the joint to such an extent as to require refinishing.

For copper, the same fluxes as for brass are used. On old work it is almost always necessary to scrape the parts to be joined to get the solder to hold. A particularly difficult piece of work to solder is an old bath tub. The grease and soap form a layer that is impervious to any of the fluxes, and it must be carefully removed entirely

if good work is wanted.

For zinc, use muriatic acid almost full strength or chloride of zinc solution. Zinc is the metal that has a "critical temperature" more than any other metal except the softer alloys. If the iron is overheated, the zinc is melted and a hole burned in the metal; even if this does not occur, the surface of the metal is roughened and there is formed on the soldering copper an alloy that will not flow but simply makes a pasty mass. At the correct heat the solder will flow readily and unite firmly with the metal. Especially if the work is to be painted, care should be taken to neutralize and wash off any excess of acid or soldering solution, as it is impossible to cause paint to adhere properly unless this is done.

For galvanized iron, use muriatic acid, chloride of zinc solution or rosin, and be sure to see that the acid is neutralized if the work is to be painted. Many cornices and fronts are made of this metal and are very unsightly in a short time after being painted, particularly at the joints, owing to lack of care in removing the excess flux.

An action that is not usually taken into consideration in the joining of galvanized iron or zinc with copper, as is sometimes done, is the electrical action set up by the metals if any moisture is present. This is very noticeable in cities where the acid from the atmosphere assists in the action. It will nearly always be found that the zine or galvanized iron has been greatly injured at the places joined.

For wrought iron or steel, zinc chloride is best. The iron or steel, to make good work, should be previously freed from scale or oxide

and tinned before joining. Where the oxide is not very heavy, the iron can be cleaned by brushing with muriatic acid and rubbing with a piece of zinc.

The Fluxes Themselves

The above paragraphs give the fluxes adapted to the various metals; the fluxes themselves are as follows:

Hydrochloric or muriatic acid. The ordinary commercial acid is much used in full strength or slightly diluted to solder zinc, par-

ticularly where the zinc is old or covered with an oxide.

Rosin or colophony, powdered, is commonly used for copper, tin and lead, and very generally by canneries and packing houses on account of its non-poisonous qualities. It is also used mixed with common olive oil. Turpentine can also be used as a flux. Beeswax is a good but expensive flux. Tallow is also used for lead pipe, but is more frequently mixed with rosin.

Palm or cocoa oil will work well, but is more generally used in the manufacture of tin plate. The common green olive oil works

very well with the more fusible solders.

As expedients we can use a piece of common stearine candle or a piece of common brown rosin soap or cheap furniture varnish, which is largely composed of rosin. Paraffin, vaseline and stearine are recommended for use with some of the alloys for soldering aluminum.

Chloride of zinc, acid, or soldering liquid, is the most commonly used of all the fluxes; as usually prepared, simply dissolve as much scrap zinc in the ordinary commercial muriatic acid as it will take up. But if it is diluted with an equal quantity of water and a small quantity of sal ammoniac is added it works much better and is less likely to rust the articles soldered. If they are of iron or steel, about 2 ounces to the pint of solution is about the proper quantity of powdered sal ammoniac to add.

In preparing this solution, a glass or porcelain vessel should be used; owing to the corrosive fumes, it should be done in a well-ventilated place. Use a vessel of ample capacity, since there is con-

siderable foaming or boiling of the mixture.

Soldering liquid, non-corrosive, is also prepared by dissolving the zinc in the acid as above and adding one fourth of the quantity of aqua ammonia to neutralize the acid, then diluting with an equal quantity of water.

Soldering liquid, neither corrosive nor poisonous. Dissolve 12

parts glycerin, 12 parts water, and add 11 parts lactic acid.

Soldering paste. When a solution of chloride is mixed with starch paste, a syrupy liquid is formed which makes a flux for soldering.

Soldering fat or paste. Melt I pound of tallow and add I pound of common olive oil. Stir in 8 ounces of powdered rosin; let this boil up and when partially cool, add with constant stirring, 1 pint of water that has been saturated with powdered sal ammoniac. Stir constantly until cool. By adding more rosin to make it harder, this can be formed into sticks. A very good acid mixture for cleaning work to be soldered is equal parts of nitric and sulphuric acid and water. Never pour the water into the acid.

Cleaning and Holding Work

For copper work a dilute sulphuric acid is best. Articles of lead and zinc can be cleaned with a potash solution, but care must be exercised as the alkalies attack these metals. For zinc, a dilute solution of sulphuric or muriatic acid will clean the surface.

For cleaning or removing the oxide or other foreign material, scrapers and files are frequently used. An old file bent at the ends and with the corners flaring makes a handy tool. Grind the edge

sharp and make as hard as possible.

To enable difficult points to be "filled," sometimes a small piece of moist clay pressed into shape to form the desired shape can be used to advantage as a guide for the solder. Another use of the clay is to embed the parts in, to hold them in position for soldering.

Plaster of paris is also used for this purpose, but is sometimes difficult to remove, especially in hollow pieces. A dilute solution

of muriatic acid will help to get this out, however.

Castings containing aluminum are always harder to solder than other alloys. In some instances where the percentage of aluminum is high, it is necessary to copperplate the parts to be joined before a satisfactory joint can be made. In nearly every instance the work

can be "stuck" together, but not actually soldered.

In metal-pattern making too little attention is given to both the fitting of the parts and the selection of the solder to joint the work. A good grade should always be used, and it must be borne in mind that the higher the melting point of the solder the stronger the joint.

A very good job of soldering can be done on work that will permit of it by carefully fitting the parts, laying a piece of tin foil, covered on both sides with a flux, between the parts to be joined and pressing them tightly together. Heat until the foil is melted. This is very good in joining broken parts of brass and bronze work. If they fit well together, they can frequently be joined in this manner so that the joint is very strong and almost imperceptible.

Soldering Cast Iron

For cast iron, the flux is usually regarded as a secret. A number of methods are in use; one of the oldest and least satisfactory is to brush the surfaces thoroughly with a brass scratch brush. Brush until the surface is coated with brass, then tin this surface and solder as usual. If plating facilities are to be had, copperplate the parts and solder together. This method has been used very successfully for a number of years.

A fair substitute for the above is to clean the surfaces thoroughly and copperplate them with a solution of sulphate of copper; about I ounce sulphate of copper, ½ pint water, ½ ounce sulphuric acid. Brush this solution on or dip into the solution, rinse off and dry it

well before soldering.

Another method is to tin the cast iron. To do this, first remove all scale until the surface is clean and bright. The easiest way to do this is with the emery wheel. Dip in a lye to remove any grease, and rinse the lye off; then dip into muriatic acid of the usual strength.

Then go over the surface with rosin and a half and half solder. It may be necessary to dip into the acid several times to get the piece thoroughly tinned. Rubbing the surface of the iron with a piece

of zinc while the acid is still on it will facilitate the tinning.

Another method of soldering cast iron is to clean the surface as in the previous operation and then brush over with chloride of zinc solution and sprinkle powdered sal ammoniac on it; then heat until the sal ammoniac smokes. Dip into melted tin and remove the surplus; repeat if not thoroughly tinned. Half tin and half lead works well as a solder for this.

Commutator wires and electrical connections should never be soldered by using an acid solution, owing to the corrosive action after-

ward. A good flux is an alcoholic solution of rosin.

Cold Soldering for Metals, Glass, Porcelain, etc.

Precipitate the copper from a solution of the sulphate by putting in strips of zinc. Place the copper powder in a porcelain or wedge-wood mortar and mix it with from 20 to 30 parts of sulphuric acid of 1.85 degrees Baumé. Then add 70 parts of mercury when well mixed, wash well with water to remove the excess of acid and allow it to cool. To use it, heat it and pound it well in an iron mortar until it becomes plastic. It can then be used and adheres very firmly when cold. No flux is needed, but the surfaces must be clean. This is used where heat cannot be used, or to join metal parts to glass or porcelain.

A solution of copper for copperplating steel or cast iron before soldering will work by simply immersing the work in it. This is also useful to copper the surface of dies and tools to enable the mechanic to "lay out" or scribe the work so that the lines can be readily seen. Take copper sulphate $3\frac{1}{2}$ ounces, sulphuric acid $3\frac{1}{2}$ ounces, water 1 to 2 gallons. Dissolve the copper in the water and add the

acid.

Solders and Fusible Alloys

Solders act under constant stress considerably like plastic or semi-fluid material. Their fluidity resembles that of tar or gum, and their distortion with time is greater than would be thought. In a series of tests a notable point brought out was the varying degrees of strength with age. Tensile strength increases with the percentage of tin present, but when the solder's age is considered as a factor, the product possesses its maximum value at 60 per cent. tin, showing this property as similar to that of the melting point and depending upon chemical composition.

For general work, the solder requiring resistance to stress is 60 per cent. tin, but for work requiring little mechanical strength, such

as sealing, a lower per cent. of tin may be used.

Generally speaking, all solders are alloys of lead and tin. The more lead the alloy contains, above 40 per cent., the higher is its melting point, as also the less lead it contains, below 40 per cent., the lower is its melting point.

The melting point of alloys which fuse at a low temperature may be found by tying a small wire around a fragment of alloy and hanging it in a bath of water. A thermometer should be kept in the bath and the temperature increased slowly until the alloy melts. ing point of the alloy can then be noted by the temperature of the bath. For higher temperature a bath of paraffin or oil is used. bismuth is added to these alloys the melting point is lower, as bismuth possesses the quality of expanding on cooling, a property which is very unusual in metals. Bismuth is used not only to make the alloy or solder more easily worked, by diminishing its melting point, but if sufficient quantity be present its expansive tendency counterbalances the effects of the contraction of the other metals, and the total result is the prevention or reduction of shrinkage in the mold. The addition of cadmium still farther lowers the melting point of such allovs as those of bismuth, lead and tin, which in themselves have very low melting points.

Composition and Melting Point of Solders and Fusible $$\operatorname{Alloys}$$

				Other	Melting	Point
Alloy	Lead	Tin	Bis- muth	Constitu- ents	Cent.	Fahr.
Solder I Solder 2 Solder 3 Solder 4 Solder 5 Solder 6 Solder 7 Solder 8 Solder 9 I. Steam boiler plug 2 Steam boiler plug 3 Steam boiler plug 4 Steam boiler plug Gir Isaac Newton's. Suitable for casts Rose's alloy D'Arcet's alloy Wood's alloy Lipowitz's alloy Expanding alloy	96.15 90.9 83.3 75.0 66.7 50.0 40.0 33.3 33.3 48.4 44.5 42.1 10.0 30.0 31.25 28.1 25.0 26.9 66.7	3.85 9.1 16.7 25.9 33.3 50.0 66.0 66.7 33.3 40.0 20.0 18.75 21.9 25.0 12.5 12.7	33.3 12.8 22.2 42.1 50.0 50.0 50.0 50.0 50.0 50.0 8.3	Zinc 38.8 33.3 15.8 Cadmium 12.5 10.4 25.0	292 283 266 250 227 188 168 171 140 171 141 123 116 100 98 95 93 60 66 66	558 541 511 482 441 370 284 340 284 340 285 253 240 212 208 203 200 140 150

Hard solders sometimes contain more or less copper. A substantial solder contains 60 per cent. copper, 20 per cent. tin and 20 per cent. zinc. An easily melted yellow, hard solder contains about 45 per cent. copper and 55 per cent. zinc. This solder is really a brass, but at times is used for soldering, binding and filling purposes.

Nearly all aluminum solders are alloys of tin and aluminum that contain from 15 to 25 per cent. aluminum. A small per cent. of copper or nickel, never exceeding 2 or 3 per cent., is sometimes used. The exact point of separation between a fusible metal and a nonfusible one is very uncertain, thus several additional alloys are given in the table. In filling up imperfections in ornamental castings for plugs in electrical wiring and on boilers in engineering work, fusible alloys are used. Sometimes defects in structural steel have been filled in with expanding alloy, after being dressed in a coat of point. The United States Government rules call for pure Banca tin for boiler plugs, but this is not essential and any good tin will serve the purpose.

Experiments have been made in an engineering college quite recently for the purpose of finding the way of making solder joints, as well as measuring their tensile strength. Any pressure upon the solder at the moment of setting diminishes the strength of the joint. Thus, in making a solder joint, the upper piece should be held above the lower one, the solder fused by means of two blow torches, and the pieces brought together by very slow and easy pressure. By employing this method, which differs from that commonly called "sweating," the joint is less liable to be broken, as the crystalline composition of

the resulting mass contains less resistance at this time.

In addition it is found that there is remarkable variation with time of the tensile strength of such joints, which is also in accord with what would be considered proper by engineering science in this field. Under any circumstances the average strength attained does not exceed 27,000 pounds per square inch, and was obtained from solder made with three-fifths of its composition tin.

ARSENAL "HOT DIP" PROCESS FOR TINNING

Use a metal composed of 80 parts of lead to 20 parts of tin by weight. The steel plate to be tinned is first pickled in a bath of 40 parts of water to one part of sulphuric acid by volume. After pickling, the metal is washed in clean water to remove all traces of the pickling acid. The work is then dipped in a flux which is made by dissolving zinc in hydrochloric acid until it is saturated. After dipping in the flux, the pieces are dipped in the melted metal (80% lead — 20% tin) until thoroughly coated and are then shaken off and thrown in a pile to cool.

A Method of Tinning Brass Parts

Brass parts are placed in layers in a screened basket with tin plates between each layer. They are then placed in a copper tank filled with water supplied with steam coil and brought to a boiling point. Sufficient amount of Cream of Tartar is added until the parts are properly plated. Four hours are required to properly tin these parts.

Small parts are placed in a cheese cloth bag in a solution of one pint of phosphoric acid (U.S.P. 50%) to four gallons of water. Heat to a boiling point until pins begin to turn white, requiring about two hours. Remove and place in linseed oil. They are then

rolled on staw boards to remove the surplus oil.

GEARING

GEAR TEETH - SHAPES OF

Cycloidal or Epicycloidal. — A curved tooth generated by the point of a circle rolling away from the gear wheel or rack.

Involute. - A curved tooth generated by unwinding a tape or

string from a cylinder. The rack tooth has straight sides.

Involute Standard. — The standard gear tooth has a $14\frac{1}{2}$ degree pressure angle which means that the teeth of a standard rack have straight sides $14\frac{1}{2}$ degrees from the vertical.

Involute - Stubbed. - A tooth shorter than the standard and

usually with a 20-degree pressure angle.

GEARS — TEETH AND PARTS

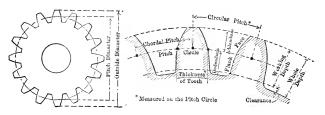


Fig. 1. - Part of Gear Teeth

Addendum. — Length from pitch line to outside.

Chordal Pitch. — Distance from center to center of teeth in a straight line.

Circular Pitch. — Distance from center to center of teeth measured on the pitch circle.

Clearance. - Extra depth of space between teeth.

Dedendum. — Length from pitch line to base of tooth.

Diametral Pitch. Number of teeth divided by the pitch diameter or the teeth to each inch of diameter.

Face. — Working surface of tooth outside of pitch line. Flank. — Working surface of tooth below pitch line.

Outside Diameter. — Total diameter over teeth.

Pitch Diameter. — Diameter at the pitch line.

Pitch Line. — Line of contact of two cylinders which would have the same speed ratios as the gears.

Linear Pitch. — Sometimes used in rack measurement. Same as

circular pitch of a gear.

Having	To Get	Rule	Formula
The Diametral Pitch	The Circular Pitch	Divide 3.1416 by the Diametral Pitch	$P' = \frac{3.1416}{P}$
The Pitch Diameter and the Number of Teeth	The Circular Pitch	Divide Pitch Diameter by the product of .3183 and Number of Teeth	l .
The Outside Diameter and the Number of Teeth	The Circular Pitch	Divide Outside Diameter by the product of .3183 and Number of Teeth plus 2	$P' = \frac{D}{.3183 N + 2}$
The Number of Teeth and the Circular Pitch	Pitch Diameter	The continued product of the Number of Teeth, the Circular Pitch and .3183	D' = NP' .3183
The Number of Teeth and the Outside Diameter	Pitch Diameter	Divide the product of Number of Teeth and Outside Diameter by Number of Teeth plus 2	$D' = \frac{ND}{N+2}$
The Outside Diameter and the Circular Pitch	Pitch Diameter	Subtract from the Outside Diameter the product of the Circular Pitch and .6366	D' = D - (P'.6366)
Addendum and the Number of Teeth	Pitch Diameter	Multiply the Number of Teeth by the Addendum	D' = N s
The Number of Teeth and the Circular Pitch	Outside Diameter	The continued product of the Number of Teeth plus 2, the Circular Pitch and .3183	$D = (N+2) \\ P' .3183$
The Pitch Diameter and the Circular Pitch	Outside Diameter	Add to the Pitch Diameter the product of the Cir- cular Pitch and .6366	D = D' + (P'.6366)
The Number of Teeth and the Addendum	Outside Diameter	Multiply Addendum by Number of Teeth plus 2	D = s (N+2)
The Pitch Diameter and the Circular Pitch	Number of Teeth	Divide the product of Pitch Diameter and 3.1416 by the Circular Pitch	$N = \frac{D'3.1416}{P'}$
The Circular }	Thickness of Tooth	One half the Circular Pitch	$t = \frac{P'}{2}$
The Circular }	Addendum	Multiply the Circular Pitch by .3183 or $s = \frac{D'}{N}$	s = P'.3183
The Circular }	Root	Multiply the Circular Pitch by .3683	s + f = P' .3683
The Circular }	Working Depth	Multiply the Circular Pitch by .6366	D'' = P' .6366
The Circular }	Whole Depth	Multiply the Circular Pitch by .6866	
The Circular }	Clearance	Mutliply the Circular Pitch by .05	f = P' .05
Thickness of }	Clearance	One tenth the Thickness of Tooth at Pitch Line	$f = \frac{t}{10}$

Having	To Get	Rule	Formula
The Circular }	The Diametral Pitch	Divide 3.1416 by the Circular Pitch	$P = \frac{3.1416}{P'}$
The Pitch Di- ameter and the Number of Teeth	The Diametral Pitch	Divide Number of Teeth by Pitch Diameter	$P = \frac{N}{D'}$
The Outside Diameter and the Number of Teeth The Number of	The Diametral Pitch	Divide Number of Teeth plus 2 by Outside Di- ameter	$P = \frac{N+2}{D}$
The Number of Teeth and the Diametral Pitch	Pitch Diameter	Divide Number of Teeth by the Diametral Pitch	$D' = \frac{N}{P}$
The Number of Teeth and the Outside Di- ameter	Pitch Diameter	Divide the Product of Outside Diameter and Number of Teeth by Number of Teeth plus 2 Subtract from the Outside	$D' = \frac{DN}{N+2}$
The Outside Diameter and the Diametral Pitch	Pitch Diameter	Subtract from the Outside Diameter the quotient of 2 divided by the Diametral Pitch	$D' = D - \frac{2}{P}$
Addendum and the Number of Teeth	Pitch Diameter	Multiply Addendum by the Number of Teeth	D' = s N
of Teeth The Number of Teeth and the Diametral Pitch	Outside Diameter	Divide Number of Teeth plus 2 by the Diametral Pitch	$D = \frac{N+2}{P}$
The Pitch Diameter and the Diametral Pitch	Outside Diameter	Add to the Pitch Diameter the quotient of 2 divided by the Diametral Pitch	$D = D' + \frac{2}{P}$
The Pitch Di- ameter and the Number of Teeth	Outside Diameter	Divide the Number of Teeth plus 2, by the quotient of number of Teeth divided by Pitch Diameter	$D = \frac{N+2}{\frac{N}{D'}}$
The Number of \ Teeth and \ Addendum	Outside Diameter	Multiply the Number of Teeth plus 2 by Adden- dum	D = (N+2) s
The Pitch Diameter and the Diametral Pitch	Number of Teeth	Multiply Pitch Diameter by the Diametral Pitch	N=D'P
The Outside Diameter and the Diametral Pitch	Number of Teeth	Multiply Outside Diameter by the Diametral Pitch and subtract 2	
The Diametral }	Thickness of Tooth	Divide 1.5708 by the Diametral Pitch Divide 1 by the Diametral Pitch or $s = \frac{D'}{N}$	$t = \frac{1.5708}{P}$
The Diametral }	Addendum	Pitch or $s = \frac{D'}{M}$	$s = \frac{1}{P}$
The Diametral }	Root	Divide 1.157 by the Diametral Pitch	$s + f = \frac{1.157}{P}$
The Diametral }	Working Depth	Divide 2 by the Diametral	$D'' = \frac{2}{P}$
The Diametral }	Whole Depth	Divide 2 by the Diametral Pitch	$D'' + f = \frac{2.157}{P}$
The Diametral Pitch	Clearance	Divide .157 by the Diametral Pitch Divide Thickness of Tooth at pitch line by 10	$f = \frac{.157}{P}$
Thickness of }	Clearance	Divide Thickness of Tooth at pitch line by 10	$f = \frac{\iota}{\iota_0}$

TABLE OF CORRESPONDING DIAMETRAL AND CIRCULAR PITCHES

Table No. 1		TABLE No. 2
Circular Pitch	Circular Pitch	Diametral Pitch
2.5133 2.0944 1.7952 1.571 1.396 1.257 1.142 1.047 .898 .785 .628 .524 .449 .393 .349 .314 .286 .262 .224 .196 .175 .157 .143 .131	2	1.571 1.676 1.795 1.933 2.094 2.185 2.285 2.394 2.513 2.646 2.793 2.957 3.142 3.351 3.590 3.867 4.189 4.570 5.027 5.585 6.283 7.181 8.378 10.053 12.566 16.755 25.133 50.266
.121 .112 .105 .098 .087	14 3 116 8 16	16.755 25.133
	Circular Pitch 2.5133 2.0944 1.7952 1.571 1.396 1.257 1.142 1.047 8.98 .785 .628 .524 .449 .393 .314 .286 .262 .224 .196 .175 .157 .143 .131 .121 .112 .105 .098	Circular Pitch 2.5133 2.0944 1.7952 1.571 1.396 1.257 1.142 1.047 1.898 1.047 1.898 1.14 1.785 1.898 1.14 1.785 1.898 1.14 1.785 1.142 1.047 1.15 1.36 1.36 1.36 1.36 1.36 1.36 1.36 1.36

No. 1 table shows the diametral pitches with the corresponding circular pitches.

No. 2 table shows the circular pitches with the corresponding

diametral pitches.

It is most natural to think of gears in circular or linear pitch and we soon get to know the size of any pitch, as 12, as being a little over 1 inch from center to center. But the diametral system has many advantages in figuring gear blanks, center distances, etc.

The Center Distance of any pair of spur gears is found by adding one-half the pitch diameter of both gears.

CONSTANTS FOR DETERMINING CHORDAL PITCH AND RADIUS OF SPUR GEARS

P =Chordal Pitch of Teeth.

R = Radius of Pitch Circle.

N = Number of Teeth.

C = Constant. (See table below.)

Radius of pitch circle

Chordal pitch = $\frac{\text{Radius of pitch circle}}{\text{Constant for number of teeth}}$

Radius of pitch circle = Constant \times chordal pitch.

Constant for any number of teeth = $\frac{\text{Radius of pitch circle}}{\text{Chordal pitch of teeth}}$

EXAMPLES: 1. What is radius of pitch circle of a gear having 45 teeth, 1\frac{1}{4} inch pitch? Follow 40 in table to column 5 (making 45 teeth), and find 7.168. Multiply by pitch, 1\frac{2}{4} inch, and get 12.54 inches radius or 25.08 pitch diameter.

2. What is the chordal pitch of a gear 32 inches pitch diameter, 67 teeth? Follow 60 in table to column 7 and find 10.668. Divide radius ($\frac{1}{2}$ of 32 = 16 inches) by constant. $16 \div 10.668 = 1.5$ inch

pitch.

3. What number of teeth has a gear of 1.5 inch chordal pitch and pitch diameter 32 inches? Divide by 2 to get radius. Divide this by chordal pitch which will give constant. $16 \div 1.5 = 10.666$. Look in table for this constant which will be found to represent 67 teeth.

TABLE OF CONSTANTS

N	О	1	2	3	4	5	6	7	8	9
0	0.000	0.000	0.500	0.577	0.707	0.851	1.000	1.152		
10 20	1.618 3.196	1.774 3.355	1.932 3.513	2.089 3.672	2.247 3.831	2.405 3.989		2.721 4.307	2.879 4.465	3.038 4.624
30	4.783	4.042	5.101	5.260	5.419	5.578	5.737	5.896		6.214
40	6.373	6.532	6.601	6.850	7.000	7.168	7.327	7.486		
50	7.963	8.122	8.281	8.440		8.758	8.918	9.077	9.236	
60	9.554	9.713	9.872	10.031	10.100	10.349	10.508	10.668		10.986
70	11.145	11.304	11.463	11.622	11.781	11.040	12.000	12.258	12.418	12.577
80	12.736	12.895	13.054	13.213	13.372	13.531	13.690	13.849		
90	14.327	14.486	14.645	14.804	14.963			15.441	15.600	15.759
100	15.918	16.077	16.236	16.395	16.554	16.713		17.032		17.350
110	17.509	17.668		17.987	18.146					
120	19.101		19.419	19.579	19.738	19.897	20.056			
130	20.693			21.170	21.330	21.489		21.807		
140	22.285	22.444	22.603	22.762	22.921	23.081	23.240	23.399		
150	23.877	24.036		24.354	24.513	24.672	24.832			
160	25.468	25.627	25.787	25.946	26.105	26.264			26.742	
170	27.060		27.378	27.538	27.697	27.856	28.015	28.174		28.493
180	28.652	28.811	28.970	29.129	29.289	29.448	29.607			
190	30.242	30.403	30.562	30.721	30.880	31.040	31.199	31.358		31.676
200	31.830	31.989		32.307	32.446	32.625	32.785	32.944		33.262
210	33-427	33.586		33.905	34.064	34.223	34.382	34.542		
220	35.019	35.178	35.337	35.497	35.656	35.815	35.974	36.133	36.293	
230	36.611	36.770	36.929	37.088	37.248	37.407	37.566	37.725	37.884	38.044
240	38.203	38.362	38.521	38.680	38.839	38.999	39.158	39.317	39.476	39.635
250	39.795				1 1					
								- 1		

GEAR WHEELS

TABLE OF TOOTH PARTS — DIAMETRAL PITCH IN FIRST COLUMN

Diametral Pitch	Circular Pitch	Thickness of Tooth on Pitch Line	Addendum and $\frac{1''}{P}$	Working Depth of Tooth	Depth of Space below Pitch Line	Whole Depth of Tooth
P	P'	t	s	D"	s+f	D"+f
1234 1141234 22442234 312 245678 9011	6.2832 4.1888 3.1416 2.5133 2.0944 1.7952 1.5708 1.3963 1.2566 1.1424 1.0472 .8976 .7854 .6283 .5236 .4488 .3927 .3491 .3142 .2856	3.1416 2.0944 1.5708 1.2566 1.0472 .8976 .7854 .6981 .6283 .5712 .5236 .4488 .3927 .3142 .2618 .2244 .1963 .1745 .1571 .1428	2.0000 1.3333 1.0000 .8000 .8000 .6666 .5714 .5000 .4444 .4000 .3636 .3333 .2857 .2500 .2000 .1666 .1429 .1250 .1111 .1000 .0909	4.0000 2.6666 2.0000 1.6000 1.3333 1.1429 1.0000 .8888 .8000 .7273 .6666 .5714 .5000 .4000 .3333 .2857 .2500 .2222 .2000 .1818	2.3142 1.5428 1.1571 .9257 .7714 .6612 .5785 .5143 .4628 .4208 .3857 .3306 .2893 .2314 .1928 .1653 .1446 .1286 .1157	4.3142 2.8761 2.1571 1.7257 1.4381 1.2326 1.0785 .9587 .8628 .7844 .7190 .6163 .5393 .4314 .3595 .3081 .2696 .2397 .2157
12 13 14	.2618 .2417 .2244	.1309 .1208 .1122	.0833 .0769 .0714	.1666 .1538 .1429	.0964 .0890 .0826	.1798 .1659 .1541

To obtain the size of any part of a diametral pitch not given in the table, divide the corresponding part of r diametral pitch by the pitch required.

As it is natural to think of gear pitches as the distance between teeth the same as threads, it is well to fix in the mind the approximate center distances of the pitches most in use. Or it is easy to remember that if the diametral pitch be divided by $3\frac{1}{7}$ we have the teeth per inch on the pitch line. By this method we easily see that in a 10 diametral pitch gear there are approximately 3 teeth per inch while in a 22 diametral pitch there will be just 7 teeth to the inch.

TABLE OF TOOTH PARTS — Continued
DIAMETRAL PITCH IN FIRST COLUMN

	1	I	1	1	43	
Diametral Pitch	Circular Pitch	Thickness of Tooth on Pitch Line	Addendum and $\frac{1''}{P}$	Working Depth of Tooth	Depth of Space below Pitch Line	Whole Depth of Tooth
P	P'	t	s	D"	s+f	D"+f
15 16 17 18 19 20 22 24 26 30 32 34 36 38 40 44 46 48 56 60	.2094 .1963 .1848 .1745 .1653 .1571 .1428 .1309 .1208 .1122 .1047 .0982 .0924 .0873 .0827 .0785 .0748 .0714 .0683 .0654 .0561	.1047 .0982 .0924 .0873 .0827 .0785 .0714 .0654 .0604 .0561 .0524 .0491 .0462 .0436 .0413 .0393 .0374 .0357 .0341 .0327	.0666 .0625 .0588 .0526 .0500 .0455 .0417 .0385 .0357 .0333 .0312 .0294 .0278 .0263 .0250 .0238 .0227 .0217 .0208 .0200 .0178 .0166	.1333 .1250 .1176 .1111 .1053 .1000 .0909 .0833 .0769 .0714 .0666 .0625 .0588 .0555 .0526 .0500 .0476 .0455 .0417 .0400	.0771 .0723 .0681 .0643 .0609 .0579 .0526 .0482 .0445 .0413 .0386 .0362 .0340 .0289 .0275 .0263 .0252 .0241 .0231	.1438 .1348 .1269 .1198 .1135 .1079 .0980 .0898 .0829 .0770 .0719 .0674 .0634 .0599 .0568 .0539 .0514 .0490 .0469 .0449 .0431 .0385 .0360
=	-324	1	1	.0333	1 93	1,00

To obtain the size of any part of a diametral pitch not given in the table, divide the corresponding part of I diametral pitch by the pitch required.

As it is natural to think of gear pitches as the distance between teeth the same as threads, it is well to fix in the mind the approximate center distances of the pitches most in use. Or it is easy to remember that if the diametral pitch be divided by $3\frac{1}{7}$ we have the teeth per inch on the pitch line. By this method we easily see that in a 10 diametral pitch gear there are approximately 3 teeth per inch while in a 22 diametral pitch there will be just 7 teeth to the inch.

Gear Wheels

Table of tooth parts — circular pitch in first column

Circular Pitch	Threads or Teeth per Inch Linear	Diametral Pitch	Thickness of Tooth on Pitch Line	Addendum and Module	Working Depth of Tooth	Depth of Space below Pitch Line	Whole Depth of Tooth	Width of Thread- Tool at End	Width of Thread at Top
P'	<u>1"</u> P'	P	t	s	D"	s+f	D"+f	Р′×.31	P'×.335
$\begin{array}{cccccccccccccccccccccccccccccccccccc$	12 8 5 4 5 8 6 8 6 5 7 24 6 7 4 5 6 5 9 5 6 5 7 1 1 5 6 7 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	1.5708 1.6755 1.7952 1.9333 2.0944 2.1855 2.2848 2.3562 2.3366 2.5133 2.6456 3.510 3.5904 3.8666 3.9270 4.1888 4.5696 4.7124 5.0265 5.0265 5.2360	1.0000 .9375 .8750 .8125 .7500 .7187 .6866 .6562 .6250 .5312 .5000 .4687 .4000 .3750 .3437 .3333 .3125 .3000	.6366 .5968 .5570 .5173 .4775 .4576 .4377 .4244 .4178 .3979 .3780 .3581 .2984 .2785 .2586 .2387 .2189 .2189 .2189 .2189	1.2732 1.1937 1.1141 1.0345 .9549 .9151 .8784 .8488 .8386 .7958 .7560 .7162 .6764 .6366 .5968 .5570 .5173 .4274 .4377 .4244 .3979 .3820	.7366 .6966 .6445 .5985 .5525 .5294 .5064 .4910 .4834 .4604 .4374 .4143 .3913 .3683 .3453 .3223 .2994 .2762 .2532 .2455 .2301 .2210	1.3732 1.2874 1.2016 1.1158 1.0299 .9870 .9441 .9012 .8583 .8156 .7724 .7295 .6866 .6437 .6007 .5579 .5192 .4120	.5813 .5425 .5038 .4650 .4456 .4262 .4133 .4069 .3875 .3681 .3488 .3294 .3100 .2906 .2713 .2519 .2480 .2325 .2131 .2066 .1938	.6700 .6281 .5863 .5444 .5025 .4816 .4466 .4466 .4397 .4188 .3769 .3359 .3350 .3141 .2722 .2680 .2513 .2303 .2233 .2034
$\frac{\frac{7}{9}}{16}$	1 1 1 7 9	5.4978 5.5851	.2857 .2812	.1819	.3638	.2105	.3923	.1771	.1914 .1884

To obtain the size of any part of a circular pitch not given in the table, multiply the corresponding part of r'' pitch by the pitch required.

As an example take a gear having 21 diametral pitch to find the various tooth parts. Take 1 diametral pitch and divide 3.1416 by 21 to find the corresponding circular pitch, which is .14951. The tooth thickness is $1.5708 \div 21 = .748$; the addendum is $1.\div 21 = .04761$; the working depth is $2.\div 21. = .09522$; the depth below

TABLE OF TOOTH PARTS
CIRCULAR PITCH IN FIRST COLUMN

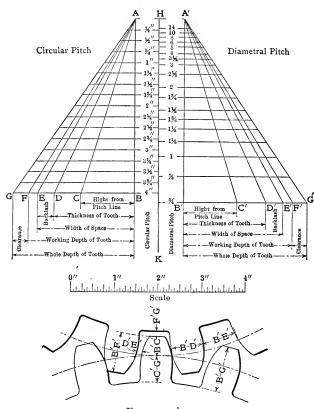
Circular Pitch	Threads or Teeth per Inch Linear	Diametral Pitch	Thickness of Tooth on Pitch Line	Addendum and Module	Working Depth of Tooth	Depth of Space below Pitch Line	Whole Depth of Tooth	Wilth of Thread- Tool at End	Width of Thread at Top
P'	<u>ı"</u>	P	t	s	D"	s+f	D"+f	P'×.31	P'×.335
12497 637 24038 411 405 6 30 24 14 29 16 31 21 16 21 17 21 16 19 17 17 18	2 2 2 2 2 2 2 2 2 3 3 3 3 4 4 5 5 5 5 6 6 7 7 8 9 10 6 20	6.2832 7.0685 7.1808 7.3304 7.8540 8.6394 9.4248 10.0531 10.4719 10.9956 12.5664 14.1372 15.7080 16.7552 17.2788 18.8496 20.4203 21.9911 23.5619 25.1327 28.2743 31.4159 50.2655 62.8318	.2500 .2222 .2187 .2143 .2000 .1875 .1818 .1666 .1562 .1500 .1429 .1250 .1111 .1000 .0937 .0939 .0714 .0666 .0625 .0555 .0500 .0312	.1415 .1393 .1364 .1194 .1158 .1061 .0995 .0955 .0909 .0796 .0707 .0537 .0537 .0489 .0455 .0425 .0398 .0318	.0796 .0707 .0637	.1842 .1637 .1611 .1578 .1473 .1381 .11228 .1151 .1105 .0921 .0818 .0737 .0690 .0614 .0566 .0526 .0409 .0368 .0230 .0184	.3433 .3052 .3003 .2942 .2746 .2578 .2289 .2146 .2060 .1962 .1716 .1526 .1373 .1287 .1249 .1144 .1055 .0981 .0917 .0858 .0763 .0687 .0429 .0343	.1550 .1378 .1356 .1328 .1240 .1163 .1127 .1033 .0969 .0930 .0886 .0775 .0689 .0620 .0581 .0504 .0517 .0477 .0443 .0310 .0310	.1675 .1489 .1466 .1436 .1340 .1218 .1117 .1047 .1047 .0057 .0838 .0744 .0670 .0528 .0515 .0479 .0446 .0419 .0372 .0335 .0209

To obtain the size of any part of a circular pitch not given in the table, multiply the corresponding part of ι'' pitch by the pitch required.

pitch line is $1.1571 \div 21 = .0551$ and the whole depth is $2.1571 \div 21 = .1027$ inches. These could also have been obtained by splitting the difference between the figures for 20 and 22 pitch. The same can be done for circular pitch except that we multiply instead of divide.

DIAGRAM FOR CAST-GEAR TEETH

The accompanying diagram (Fig. 2) for laying out teeth for cast gears will be found useful by the machinist, patternmaker and draftsman. The diagram for circular pitch gears is similar to the one given by Professor Willis, while the one for diametral pitch was obtained by using the relation of diametral to circular pitch.



Figs. 2 and 3.

By the diagram the relative size of a tooth may be easily determined. For example, if we contemplate using a gear of 2 diametral pitch, by referring to line $H\ K$, which shows the comparative distance between centers of teeth, on the pitch line, it will be observed that

2 diametral pitch is but little greater than $1\frac{1}{2}$ inches circular pitch, or exactly 1.57 inches circular pitch. This result is obtained by dividing 3.1416 by the diametral pitch (3.1416 divided by 2 equals 1.57). In similar manner, if the circular pitch is known, the diametral pitch which corresponds to it is found by dividing 3.1416 by the circular pitch; for example, the diametral pitch which corresponds to 3 inches circular pitch is by the line H K a little greater than 1 diametral pitch, or exactly 1.047 (3.1416 divided by 3 equals 1.047).

The proportions of a tooth may be determined for either diametral or circular pitch by using the corresponding diagram.

Continue, for illustration, the 2 diametral pitch. We have found, above, the distance between centers of teeth on the pitch line to be a little more than $1\frac{1}{2}$ inches (1.57 inches). The hight of tooth above pitch line B' C' will be found on the horizontal line corresponding to 2 pitch. The distance between the lines A' B' and A' C' on this line may be taken in the dividers and transferred to the scale below. Thus we find the hight of the tooth to be $\frac{1}{3}\frac{5}{2}$ inch. In the same manner the thickness of tooth B' D', width of space B' E', working depth B' F' and whole depth of tooth B' G' may be determined.

The backlash or space between the idle surfaces of the teeth of two gear wheels when in mesh is given by the distance D' E'. The clearance or distance between the point of one tooth and the bottom of space into which it meshes is given by the distance F' G'. The backlash and clearance will vary according to the class of work for which the gears are to be used and the accuracy of the molded product. For machine molded gears which are to run in enclosed cases, or where they may be kept well oiled and free from dirt, the backlash and clearance may be reduced to a very small amount, while for gears running where dirt is likely to get into the teeth, or where irregularities due to molding, uneven shrinkage, and like causes, enter into the construction, there must be a greater allowance. The diagram is laid out for the latter case. Those who have more favorable conditions for which to design gears should vary the diagram to suit their conditions. This can be done by increasing B D and decreasing B E, and by increasing B C or decreasing B G, or both. to get the clearance that will best meet the required conditions. same kind of diagram could be laid out for cut gears, but as tables are usually at hand which give the dimensions of the parts of such gears, figured to thousandths of an inch, it would be as well to consult one of these.

LAYING OUT SPUR GEAR BLANKS

Decide upon the size wanted, remembering that 12-pitch teeth are $\frac{2}{12}$ deep and 8-pitch — as in the drawing — $\frac{2}{8}$ deep, etc. Should it be 8 pitch, as shown in the cut, draw a circle measuring as many eighths of an inch in diameter as there are to be teeth in the gear. This circle is called the Pitch Line. Then with a radius $\frac{1}{8}$ of an inch larger, draw another circle from the same center, which will give the outside diameter of the gear, or $\frac{2}{8}$ larger than the pitch circle. Thus we have for the diameter of an 8-pitch gear of 24 teeth, $\frac{2}{8}$ 6. Should there be 16 teeth, as in the small spur gear in the cut, the

outside diameter would be $\frac{18}{8}$, the number of teeth being always two less than there are eighths — when it is 8 pitch — in the outside diameter.

The distance from the pitch line to the bottom of the teeth is the same as to the top, excepting the clearance, which varies from $\frac{1}{8}$ of the pitch to $\frac{1}{10}$ of the thickness of the tooth at the pitch line. This latter is used by Brown & Sharpe and many others, but the clearance being provided for in the cutters the two gears would be laid out to mesh together just $\frac{2}{8}$.

These rules apply to all pitches, so that the outside diameter of a 5-pitch gear with 24 teeth would be $\frac{2}{3}$; if a 3-pitch gear with 40 teeth it would be $\frac{4}{3}$. Again, if a blank be $4\frac{1}{8}$ ($\frac{2}{3}$) in diameter, and

cut 6 pitch, it should contain 23 teeth.

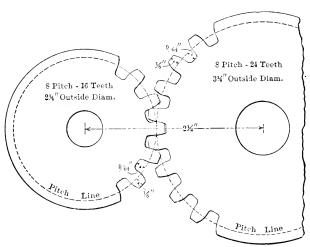
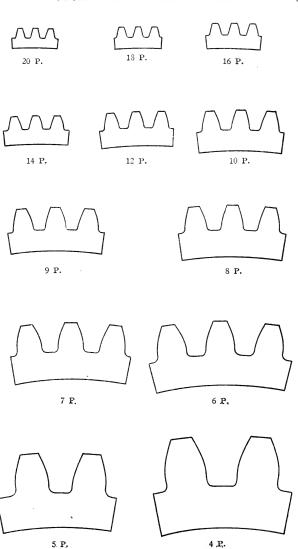


Fig. 4. - Laying out a Pair of Gears

ACTUAL SIZE OF DIAMETRAL PITCHES

It is not always easy to judge or imagine just how large a given pitch is when measured by the diametral system. To make it easy to see just what any pitch looks like the actual sizes of twelve diametral pitches are given on the following page, ranging from 20 to 4 teeth per inch of diameter on the pitch line, so that a good idea of the size of any of these teeth can be had at a glance.



LAYING OUT SINGLE CURVE TOOTH

A VERY simple method of laying out a standard tooth is shown in Fig. 5, and is known as the single curve method. Having calculated the various proportions of the tooth by rules already given, draw the pitch, outside, working depth and clearance or whole depth circles as shown. With a radius one half the pitch radius draw the semicircle from the center to the pitch circle. Take one quarter the pitch radius and with one leg at top of pitch circle strike arc cutting the semicircle. This is the center for the first tooth curve and locates the base circle for all the tooth arcs. Lay off the tooth thickness and space distances around the pitch circle and draw the tooth curves through these points with the tooth curve radius already found. The fillets in the tooth corners may be taken as one seventh of the space between the tops of the teeth.

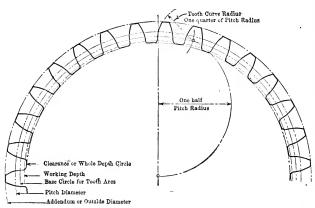


Fig. 5. — Single Curve Tooth

PRESSURE ANGLES

We next come to pressure angles of gear teeth, which means the angle at which one tooth presses against the other and can best be

shown by the pinion and rack, Figs. 6 and 7.

The standard tooth has a 14½ degree pressure angle, probably because it was so easy for the millwright to lay it out as he could obtain the angle without a protractor by using the method shown for laying out a thread tool (see Fig. 14). As the sides of an involute rack tooth are straight, and at the pressure angle from the perpendicular, draw the line of pressure at 14½ degrees from the pitch line. The base circle of the tooth arcs can be found by drawing a line from the center of the gear to the line of pressure and at right angles to it as shown, or by the first method, and working from this the tooth

curve can be drawn by either the single-curve method or, as is more usual, by stopping the curve from two or more points on this same circle.

The difference between the 14½- and 20-degree pressure angles can be seen by comparing Figs. 6 and 7. Not only is the tooth shorter, but the base is broader. The base circle for the tooth arcs is found in the same way as before.

This form of tooth is largely used in automobile transmission and similar work. William Sellers & Co. use a 20 degree pressure

angle with a tooth of standard length.

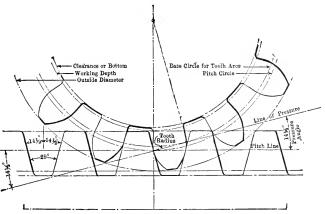


Fig. 6. - Standard Tooth

STUB-TOOTH GEARS

Any tooth shorter than the regular standard length is called a "stub" tooth, but like the bastard thread there have been many kinds. In 1899 the Fellows Gear Shaper Company introduced a short tooth with a 20-degree pressure angle instead of the usual 14½-degree. This gives a broader flank to the tooth and makes a stronger gear, especially for small pinions where strength is needed. While the Fellows tooth is shorter than the standard tooth there is no fixed relation between them, as, on account of the tooth depth graduations of the gear shaper, it was thought best to give the new tooth depth in the same scale which is shown in the following table. This means that if the pitch is 4 it has the depth of a 5-pitch standard tooth divided as shown. The clearance is one-quarter the addendum or dedendum.

TABLE OF TOOTH DIMENSIONS OF THE FELLOWS STUB-TOOTH GEAR

Cutters Marked Pitch	Stub Tooth Pitch	Has Depth of Standard Tooth	Thickness on Pitch Line	Addendum	Dedendum and Clearance
45557688799 8091102224	4 5 6 7 8 9 10	5 7 8 9 10 11 12 14	.3925 .314 .2617 .2243 .1962 .1744 .157 .1308	.200 .1429 .125 .111 .100 .0909 .0833	.250 .1785 . .1562 .1389 .125 .1137 .1042 .0893

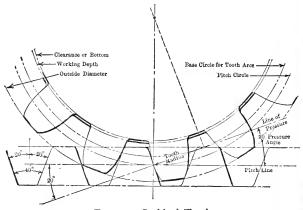


Fig. 7. — Stubbed Tooth

The Nuttall Company also use a 20-degree stub tooth, but have a fixed length or depth in the following proportions.

Addendum = $.25 \times \text{circular}$ pitch instead of .3683. Working depth = $.50 \times \text{circular}$ pitch instead of .3683. Clearance = $.50 \times \text{circular}$ pitch instead of .6366. Whole depth = $.55 \times \text{circular}$ pitch same as standard. Whole depth = $.55 \times \text{circular}$ pitch instead of .6866.

Table for Turning and Cutting Gear Blanks

FOR STANDARD LENGTH TOOTH

$\begin{array}{ c c c c c c c c c c c c c c c c c c c$	Pitch	16	12	10	1 8	Pitch	16	12	1 10	1 8
$ \begin{array}{ c c c c c c c c c c c c c c c c c c c$	Depth		.180	.216	.270	Depth of Tooth	.135	.180	1	.270
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$			Outside 1	Diameter		No. of Teeth		Outside	Diamete	r
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	of Tooth No. of Teeth 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26 27 28 29 30 31 32 33 34 35 36 37 38 39 40 41 42 43	34 1-75 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	Outside 1 I 1 1 2 1 1 2 1 1 2 1 1 1 2 1 1 2 1 1 2 1 1 2 1 1 2	Diameter 1 1 1 1 1 1 1 1 1	1 1 1 1 1 2 2 2 2 2 2 2 2 2 2 3 3 3 3 3	51 Tooth No. of Teeth 51 52 53 54 555 56 67 68 69 70 71 72 73 74 75 76 77 78 80 81 82 83 84	3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 4 4 4 4 4	Outside 41622441244125 4162255512625555555555555555555555555555	Diamete 5134000000000000000000000000000000000000	

I IO GEARING

TABLE FOR TURNING AND CUTTING GEAR BLANKS

FOR STANDARD LENGTH TOOTH

Pitch Diameters of Standard Gears

The accompanying tables require little explanation, being simply given to save time in laying out gear trains. Having the diametral pitch and the number of teeth, the pitch diameter is easily found from the tables. They also show the corresponding circular pitch, designated by C. P.

Having the pitch diameter, the outside diameter can easily be found by adding two parts of the pitch of standard gears.

The outside diameter of any gear with standard teeth is the same as the pitch diameter of two more teeth. Pitch Diameter of Gears when Diametral Pitch and the Number of Teeth are Known

D.P. C.P.	1.5 2.094	1.571	2.5	3 I.047	0.785	5 0.628	6 0.524	8	10 0.3142	0.2618	14 0.2244	16 0.1963	20 0.1571
No. T													
	10.000	7.50	00.9	2.000	3.75	3.00	2.500	1.875	1.5	1.250	1.071	0.037	0.75
16	10.666	8	6.40	5.333	00.4	3.20	2.666	2.000	9.1	I.333	I.142	1,000	0.80
11	11.222	8	6.80	2.666	4.25	3.40	2.833	2.125	1.7	1.416	1.214	1.062	0.85
× ×	12,000	000	7.20	000.9	4.50	3.60	3.000	2.250	8.1	1.500	1.285	1.125	0.00
	12.666	02.0	2.60	6.333	4.75	3.80	3.166	2.375	1.9	1.583	1.357	1.187	0.05
	T2 222	10.00	800	999.9	8	00.4	3.333	2.500	2.0	1.666	1.428	1.250	0.i
	14.000	10.50	8.40	7.000	5.25	4.20	3.500	2.625	2.1	1.750	1.500	1.312	1.05
	999 71	11.00	8.80	7.333	5.50	4.40	3.666	2.750	2.2	1.833	1.571	1.375	01.1
22	15.223	11.50	0.20	7.666	5.75	4.60	3.833	2.875	2.3	916.1	I.642	I.437	1.15
	000	12.00	0.60	8.000	0.00	4.80	4.000	3.000	2.4	2.000	1.714	1.500	1.20
	16.666	12.50	10.00	8.333	6.25	00.5	4.166	3.125	2.5	2.083	1.785	I.562	1.25
96	17.333	13.00	10.40	8.666	6.50	5.20	4.333	3.250	5.6	2.176	1.857	1.625	1.30
	1800	12.50	10.80	0.000	6.75	5.40	4.500	3.375	2:7	2.250	1.928	1.687	1.35
	18.666	14.00	11.20	0.333	7.00	2.60	4.666	3.500	2.8	2.333	2.000	1.750	1.40
	TO 222	14.50	11.60	0.000	7.25	2.80	4.833	3.625	2.0	2.416	2.07I	1.812	1.45
	00000	15.00	12.00	10.000	7.50	6.00	2.000	3.750	3.0	2.500	2.142	1.875	I.50
	999 02	15.50	12.40	10.333	7.75	6.30	5.166	3.875	3.1	2.583	2.214	I.937	1.55
	2T 222	16.00	12.80	10.666	80.80	6.40	5.333	4.000	3.2	2.666	2.285	2.000	1.60
	22.000	16 50	12.20	11.000	8.25	9.60	5.500	4.125	3.3	2.750	2.357	2.062	1.65
	999 00	17.30	12 60	TT 223	000	6.80	2.666	4.250	3.4	2.833	2.428	2.125	1.70
	22.23	17.50	14.00	11.666	8.75	7.00	5.833	4.375	3.5	2.916	2.500	2.187	1.75
	50.00	200.14	-		2					_			

ed)	20 0.1571		.80	.85	06:	.95	0	.05	01.	.15	.20	25	.30	-35	-40	.45	.50	.55	0.	.05	.70	.75	8.	.85	9	.95	8	.05	oi.	.15	.20	.25	30
tinn	~ ŏ ŏ	<u> </u>	-	_	_	_	_			··	_			_					-	-	_		_			`	(7)	(4)	-	~	,	-	
(Continued)	16 0.1963		2.250	2.312	2.375	2.437	2.500	2.503	2.025	2.087	2.750	2.812	2.875	2.937	3.000	3.002	3.125	3.187	3.250	3.312	3.375	3.437	3.500	3.502	3.625	3 687	3.750	3.812	3.875	3.937	4.000	4.002	4.125
Known.	14 0.2244		2.571	2.642	2.714	2.785	2.857	2.028	3.000	3.071	3.142	3.214	3.285	3.357	3.428	3.500	3.571	3.042	3.714	3.785	3.857	3.928	4.000	4.071	4.142	4.214	4.285	4.357	4.428	4.500	4.571	4.042	4.714
ARE	0.2618		3.000	3.083	3.176	3.250	3.333	3.416	3.500	3.553	3.000	3.750	3.833	3.916	4.000	4.083	4.170	4.250	4.333	4.410	4.500	4.583	4.000	4.750	4.833	4.916	5.000	5.083	5.176	5.250	5.333	5.410	2.500
ог Теетн	10		3.6	3.7	3.8	3.9	4.0	4.1	4.2	4.3	4.4	4.5	4.6	4.7	8.4	4.9	5.0	5.1	5.2	5.3	5.4	5.5	5.0	5.7	5.8	5.0	0.0	1.9	6.2	6.3	6.4	0.5	0.0
NUMBER	8		4.500	4.625	4.750	4.875	5.000	5.125	5.250	5.375	5.500	5.625	5.750	5.875	0.000	6.125	6.250	6.375	6.500	6.625	6.750	6.875	7.000	7.125	7:250	7.375	7.500	7.625	7.750	7.875	8.000	8.125	8.250
AND THE N	6		0.000	991.9	6.333	6.500	999.9	6.833	7.000	7.166	7.333	7.500	2.000	7.833	8.000	8.166	8.333	8.500	8.666	8.833	0.000	991.6	9.333	0.500	999.6	9.833	10.000	10.166	10.333	10.500	10.666	10.833	11,000
Ргтсн А	5 0.628		7.20	7.40	2.60	7.80	8.00	8.20	8.40	8.60	8.80	0.6	0.50	9.40	09.6	9.80	10.00	10.20	10.40	10.60	10.80	00.11	11.20	11.40	11.60	11.80	12.00	12.20	12.40	12.60	12.80	13.00	13.20
DIAMETRAL	0.785		0.00	9.25	9.50	9.75	10.00	10.25	10.50	10.75	11.00	11.25	11.50	11.75	12.00	12.25	12.50	12.75	13.00	13.25	13.50	13.75	14.00	14.25	14.50	14.75	15.00	15.25	15.50	15.75	16.00	16.25	16.50
WHEN DIA	3		12.000	12.333	12.666	13.000	13.333	13.666	1.1.000	14.333	14.666	15.000	15.333	15.666	16,000	16.333	16.666	17.000	17.333	17.666	18.000	18.333	18.666	19.000	19.333	19.666	20,000	20.333	20.666	21.00	21.333	21.666	22.000
GEARS WI	2.5		14.40	14.80	15.20	15.60	16.00	16.40	16.80	17.20	17.60	18.00	18.40	18.80	19.20	09.61	20.00	20.40	20.80	21.20	21.60	22.00	22.40	22.80	23.20	23.60	24.00	24.40	24.80	25.20	25.60	26.00	26.40
OF	2 I.57I		18.00	18.50	19.00	19.50	20.00	20.50	21.00	21.50	22.00	22.50	23.00	23.50	24.00	24.50	25.00	25.50	26.00	26.50	27.00	27.50	28.00	28.50	29.00	29.50	30.00	30.50	31.00	31.50	32.00	32.50	33.00
DIAMETER	1.5		24.000	24.666	25.333	26.000	26.666	27.333	28.000	28.666	29.333	30.000	30.666	31.333	32.000	32.666	33-333	34.000	34.666	35.333	36.000	36.666	37.333	38.000	38.666	39.333	40.000	40.666	41.333	42.000	42.666	43-333	44.000
Price 1	D.P. C.P.	No. T	36	37	38	39	40	41	42	43	44	45	46	47	48	49	50	51	52	53	54	55	26	57	58	59	9	6I	62	63	64	65	

D.P. C.P.	1.5	1.571	2.5 1.257	3 1.047	0.785	5 0.628	6	8 0.393	10	0.2618	14 0.2244	16 0.1963	20 0.1571
No. T													
	44.666	33.50	26.80	22.333	16.75	13.40	11.166	8.375	6.7	5.583	4.785	4.187	3.35
89	45.333	34.00	27.20	22.666	17.00	13.60	11.333	8.500	6.8	5.666	4.857	4.250	3.40
6g	46.000	34.50	27.60	23.000	17.25	13.80	11.500	8.625	6.9	5.750	4.928	4.312	3.45
20	40.000	35.00	28.90	23.333	17.50	14.00	11.666	8.750	7.0	5.833	5.000	4.375	3.50
71	47.333	35.50	28.40	23.666	17.75	14.20	11.833	8.875	7.1	5.916	5.071	4.437	3.55
72	48.000	36.00	28.80	24.000	18.00	14.40	12.000	000.6	7.2	0,000	5.142	4.500	3.60
73	48.666	36.50	20.20	24.333	18.25	14.60	12.166	9.125	7.3	6.083	5.214	4.562	3.65
74	49.333	37.00	29.60	24.000	18.50	14.80	12.333	9.250	7.4	6.176	5.285	4.625	3.70
75	20.000	37.50	30.00	25.000	18.75	15.00	12.500	9.375	7.5	6.250	5.357	4.687	3.75
70	50.000	38.00	30.40	25.333	19.00	15.20	12.666	0.500	9.2	6.333	5.428	4.750	3.80
77	51.333	38.50	30.80	25.000	19.25	15.40	12.833	9.625	7.7	6.416	5.500	4.812	3.85
78	52.000	30.00	31.20	20.000	19.50	15.60	13.000	9.750	7.8	6.500	5.571	4.875	3.90
79	\$2.00 0	39.50	31.60	20.333	19.75	15.80	13.100	9.875	6.7	6.583	5.642	4.937	3.95
200	53-333	40.00	32.00	20.000	20.00	10.00	13.333	10.000	0.0	999.9	5.714	5.000	4.00
SI	24.000	40.50	32.40	27.000	20.25	16.20	13.500	10.125	8.1	6.750	5.785	5.002	4.05
82	24.000	41.00	32.80	27.333	20.50	10.40	13.000	10.250	8.2	6.833	5.857	5.125	4.10
 800	55.333	41.50	33.20	27.000	20.75	16.60	13.833	10.375	8.3	6.916	5.928	5.187	4.15
84	20.000	42.00	33.00	28.000	21.00	16.80	14.000	10.500	8.4	7.000	0.000	5.250	4.20
95	50.000	42.50	34.00	28.333	21.25	17.00	14.100	10.625	10°	7.083	0.07 I	5.312	4.25
	57.333	43.00	34.40	28.000	21.50	17.20	14.333	10.750	9.8	7.r76	6.142	5.375	4.30
00	58.000	43.50	34.80	20.000	21.75	17.40	1+.500	10.875	2.7	7.250	6.214	5.437	4.35
	59.000	8.4	35.20	29.333	22.00	17.00	14.000	11.000	× ×	7.333	0.285	5.500	4.40
60	59.333	44.50	35.00	29.000	22.25	17.30	14.833	11.125	6.8	7.410	0.357	5.502	4.45
3 6	90.000	25.00	30.00	30.000	22.50	19.00	15.000	11.250	0.6	7.500	0.428	5.025	4.50
91	67,333	45.50	30.40	30.333	22.75	10.20	15.100	11.375	1.6	7.583	0.500	5.087	4.55
	65.133	40.00	30.00	30.000	23.00	10.40	15.333	11.500	9.5	2.000	0.571	5.750	4.00
	05.000	40.50	37.20	31.000	23.25	18.00	15.500	11.025	9.3	7.750	0.042	5.812	4.05
94	02.000	47.00	37.00	31.333	23.50	18.80	15.000	11.750	6.4	7.833	6.71.4	5.875	4.70
05	03.333	47.50	38.00	31.666	23.75	19.00	15.833	11.875	9.5	7.916	6.785	5.937	4.75
06	04.000	48.00	38.40	32,000	24.00	19.20	16.000	12.000	9.6	8.000	6.857	000.9	4.80
07	04.000	48.50	38.80	32.333	24.25	19.40	16.166	12.125	6.7	8.083	6.928	6.062	4.85
	05.333	40.00	39.20	32.000	24.50	10.60	16.333	12.250	8.6	8.176	7.000	6.125	4.90
66	000.00	49.50	39.60	33.000	24.75	19.80	16.500	12.375	6.6	8.250	7.071	6.187	4.95
				-	_	-		_	_	_	_	_	

PITCH DIAMETER OF GEARS WHEN DIAMETRAL PITCH AND THE NUMBER OF TEETL ARE KNOWN. (Continued)

•	
20	0.00 0.00 0.00 0.00 0.00 0.00 0.00 0.0
16	6.250 6.350 6.350 6.560 6.562 6.650 6.650 6.650 7.750 7.125
14 0.2244	7.142 7.285 7.285 7.757 7.757 7.757 7.757 7.758 8.000
0.2618	8 8 8 3 3 3 8 8 4 1 6 8 8 8 5 8 8 8 3 3 8 8 5 8 8 8 8 8 8 8 8
0.3142	0.00 0.01 0.02 0.03 0.04 0.05 0.05 0.05 0.05 0.05 0.05 0.05
8	12.500 12.502 12.502 13.502 13.503 13.503 13.503 14.503 14.503 14.503 14.503 14.503 14.503 14.503 14.503 14.503 14.503 16
6 0.524	16.666 16.835 17.000 17.000 17.000 17.000 17.000 17.000 17.000 17.000 17.000 18.000 18.000 19.0000 19.000 19.000 19.000 19.000 19.000 19.000 19.000 19.000 19.0000 19.000 19.000 19.000 19.000 19.000 19.000 19.000 19.000 19.0000 19.000
5 0.628	20.20 20.20
0.785	2 5,5 5 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6
3 1.047	33.33.33.33.33.33.33.33.33.33.33.33.33.
2.5 I.257	6 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4
1.571	6.5.5.6.5.8.8.8.8.8.8.8.8.8.8.8.8.8.8.8.
1.5	66.666 69.333 68.5080 69.333 71.333 7
D.P. C.P.	N.N.O. T

B. & S. INVOLUTE GEAR TOOTH CUTTERS

No. 1 will cut wheels from 135 teeth to a rack. No. 1 will cut wheels from 80 teeth to 134 teeth.

No. 2 will cut wheels from 85 teeth to 134 teeth.

No. 2 will cut wheels from 45 teeth to 134 teeth.

No. 3 will cut wheels from 45 teeth to 54 teeth.

No. 3 will cut wheels from 55 teeth to 54 teeth.

No. 3 will cut wheels from 55 teeth to 54 teeth.

No. 4 will cut wheels from 56 teeth to 54 teeth.

No. 4 will cut wheels from 56 teeth to 54 teeth. No. $4\frac{1}{2}$ will cut wheels from 23 teeth to 25 teeth. No. 5 will cut wheels from 21 teeth to 25 teeth. No. 51 will cut wheels from 19 teeth to 20 teeth. No. 6 will cut wheels from 17 teeth to 20 teeth. No. 62 will cut wheels from 15 teeth to 16 teeth. No. 7 will cut wheels from 14 teeth to 16 teeth. No. 71 will cut wheels from 13 teeth to 14 teeth. No. 8 will cut wheels from 12 teeth to 13 teeth.

The eight cutters represented by the whole numbers constitute the regular set of cutters generally used for each pitch of tooth. half numbers increase the set to 15 and gives teeth which are theoretically more correct. In some work special cutters are used for each gear but the 15 cutters in a set offer all that most cases require.

Table Showing Depth of Space and Thickness of Tooth in Spur Wheels, when cut with these Cutters

Pitch	Depth to be	Thickness of	Pitch	Depth to be	Thickness of
of	cut in Gear	Tooth at Pitch	of	cut in Gear	Tooth at Pitch
Cutter	Inches	Line. Inches	Cutter	Inches	Line. Inches
114-12014 2 144-12014 2 2 2 3 3 3 4 5 6 7 8 9 10	1.726 1.438 1.233 1.078 .058 .863 .784 .719 .616 .539 .431 .359 .308 .270 .240 .216	1.257 1.047 .898 .785 .697 .628 .570 .523 .448 .393 .314 .262 .224 .196	11 12 14 16 18 20 22 24 26 28 30 32 36 48	.196 .180 .154 .135 .120 .108 .098 .090 .083 .077 .072 .067 .060 .054	.143 .131 .112 .098 .087 .079 .071 .065 .060 .056 .052 .049 .044

BLOCK INDEXING IN CUTTING GEAR TEETH

Block or intermittent indexing is a method to increase the output of gear cutters by allowing the feed and cutting speed to be increased without unduly heating the work. This is done by jumping from the tooth just cut to a tooth far enough away to escape the local heating and on the following rounds to cut the intermediate teeth. While the indexing takes a trifle more time, the heat is distributed so that faster cutting can be done without heating and dulling the cutter.

The following table gives the indexing of gears from 25 to 200 teeth and is worked out for the Brown & Sharpe gear cutter but can

be modified to suit other machines.

Turns of Locking Disc	10	7	7	7	7	01	01	64	7	7	61	7	61	7	CQ	7	5	7	63	7	7	7	61	7	N	81
Second Follower		52																								
Second Driver	18	84	8	84	8	8.	240	õ	8.	8	8	8	2	8	8	72	8 4	48	8.	8	84	8	∞	8	2	84
First Follower	10/	9	9	20	9	52	20	42	8	8	20	8.	8	20	3	20	8	4	8	3	20	9	20	9	20	8
First Driver	00	100	100	100	100	8	100	10	001	100	1001	100	001	8	100	001	001	100	100	100	100	100	100	001	001	8
No. Indexed at Once	v	7	Ŋ	7	'n	0	_	S	S	1	×	S	~	6	S	0	7	'n	Ŋ	'n	7	S	7	S	7	
Teeth to be Cut		162							_										_							
Turns of Locking Disc	0	0	7	01	C1	CI.	C1	C)	7	N	7	N	(1	N	01	01	0	7	0	7	7	61	7	7	2	0 0
Second Follower	82	62	20	45	64	Š	25	40	20	67	54	8	9	70	9	22	72	200	9	74	20	20	3	99	62	28
Second Driver	84	0	8	0,20	8	× ×	200	8	9	8	84	8	8	8	8	96	8	75	õ	8	84	8	8	72	72	84
First Follower	30	3	20	δ,	9	30	0,5	88	2	9	ç,	8	02	S,	94	9	8	ည	Š	8	9	8	89	20	5	200
First Driver	100	100	100	001	8	001	001	001	001	100	100	001	001	20	100	8	100	100	100	100	100	100	100	100	100	000
No. Indexed at Once	1	Ŋ	7	v	w	7	_	Ŋ	4	S	7	v	S	3	S	9	v	9	Ŋ	S	7	Ŋ	Ŋ	v,	9	v r
Teeth to be Cut	123	124	125	126	128	129	130	132	133	134	135	136	138	140	141	143	144	145	147	148	150	152	153	154	155	156
Turns of Locking Disc	10	01	8	0	7	01	7	7	01	61	8	63	C)	0	7	7	7	N	61	7	6	6	01	C	0	01 01
Second Follower	198	300	88	20	52	92	02	94	2	9	86	44	40	89	53	9	9	44	9	20	26	46	28	28	89	6 4
Second Driver	00	84	8	20	72	8	×	8	96	8	8	8	84	9	8	96	20	84	8	8	84	90	8	9	72	929
First Follower	30	30	30	30	20	30	30	30	20	30	30	30	20	30	9	2	30	20	74	9	30	2	9	30	2	6,30
First Driver	001	100	100	100	100	100	001	100	001	100	100	100	100	100	100	100	100	100	100	100	100	100	100	100	100	901
No. Indexed at Once	v	7	Ŋ	7	3	۱۲,	~	r.	4	v	'n	10	7	'n	N	4	~	7	v	v	7	∞	ır.	00	~	1-4
Teeth to be Cut	86	87	88	8	16	92	93	94	95	96	86	66	100	102	104	105	108	110	III	112	114	FII	911	117	110	120
Turns of Locking Disc	0	0	7	2	C1	63	7	N	7	01	61	7	64	01	63	7	61	7	61	N	0	0	0	61	63	0 0
Second Follower	52	4	4	20	20	28	9	62	20	64	22	40	67	99	4	70	72	74	50	26	44	78	8	52	82	84 89
Second Driver	9	8	9	8	96	8	8	8	8	8	96	S	8	8	8	8	8	8	84	8	90	8	8	8	8	88
First Follower	30	30	20	30	30	30	30	30	30	30	50	44	30	30	46	δ,	30	30	30	30	2	30	20	30	200	20 02
First Driver	100	100	100	001	100	100	100	100	100	901	100	001	001	001	100	100	001	100	001	100	100	100	90	100	100	001
No. Indexed at Once	V	'n	4	v	4	v	7	S	v	Ŋ	4	N	Ŋ	Ŋ	Ŋ	c	Ŋ	w	7	Ŋ	4	v	3	7	· V	N 4
Teeth to be Cut	22	5	55	20	27	200	8	62	63	64	65	99	67	89	69	70	72	74	75	20	11	78	8	81	82	85
Turns of Locking Disc	14	4	4	4	4	0	4	4	4	4	4	4	4	4	4	4	4	4	4	4	4	4	4	4	. 4	4 0
Second Follower	90	52	54	26	200	8	62	6.4	4	89	20	4	74	20	78	8	82	84	86	88	30	02	9	90	8	98
Second Driver	72	8	8	8	8	8	8	8	8	8	96	8	8	8	8	8	8	8	8	8	2	8	8	8	8	840
First Follower	S.	5,05	20	50	20	20	50	20	20	50	50	48	30	30	30	50	30	30	30	30	20	30	30	30	8	3 22
First Driver	100	001	100	100	100	8	100	100	100	100	100	100	100	100	100	100	100	100	100	100	100	100	100	100	100	8 8
No. Indexed at Once	<u>.</u>	٠,			_	_			_		_			_	_	_		_			_					1-4
Teeth to be Cut	0	26	27	28	29	စ္တ	31	32	33	34	35	36	37	38	30	9	41	42	43	4	4	40	47	84	40	S 12

THE DIMENSIONS OF GEARS BY METRIC PITCH

Module is the pitch diameter in mm. divided by the number of teeth in the gear.

Pitch diameter in mm. is the Module multiplied by the number of teeth in the gear.

M = Module.

D' = The pitch diameter of gear.

D =The whole diameter of gear.

N =The number of teeth in gear.

D'' = The working depth of teeth. t = Thickness of teeth on pitch line.

f = Amount added to depth for clearance.

$$M = \frac{D'}{N}$$
 or $\frac{D}{N+2}$.

$$D' = N M.$$

$$D = (N + 2) M.$$

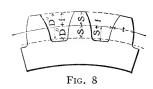
$$D = (N + 2) M.$$

$$N = \frac{D'}{M} \text{ or } \frac{D}{M} - 2.$$

$$D'' = 2 M.$$

$$t = M \text{ 1.5708.}$$

$$f = \frac{M \cdot 1.5708}{10} = .157 M.$$



The Module is equal to the part marked "S" in cut, measured in mm. and parts of mm.

Example: Module = 3.50 mm. 100 teeth.

Pitch diameter = $3.5 \times 100 = 350$ mm.

Whole diameter = $(100 + 2) \times 3.5 = 357$ mm.

PITCHES COMMONLY USED - MODULE IN MILLIMETERS

Module	Corresponding English Diametral Pitch	Module	Corresponding English Diametral Pitch
mm. 1 1.25 1.5 1.75 2 2.25 2.75 3 3.5 4	50.800 33.867 25.400 20.320 16.933 14.514 12.700 11.288 10.160 9.236 8.466 7.257 6.350	4.5 mm. 5 5.5 6 7 8 9 10 11 12 14 16	5.644 5.080 4.618 4.233 3.628 3.175 2.222 2.540 2.309 2.117 1.814 1.587

SPROCKET WHEELS FOR BLOCK CENTER CHAINS

N = No. of Teeth.

C = Diameter of Round Part ofChain Block. Tan D =

Sin. E+ Cos. E

B =Center to Center of holes in Chain Block.

A =Center to Center of holes in side

links. Pitch Diam. = $\frac{A}{\sin D}$.

Outside Diam. = Pitch Diam. + C. Bottom Diam. = Pitch Diam. -C.

Fig. 9 In calculating the diameter of Sprocket Wheels the Bottom Diameter is the most important. DIAMETER OF SPROCKET WHEELS - FOR BLOCK CHAINS I" PITCH A = .6". B = .4". C = .325".

	T = 0. $D =$.4. 0325 .	
No. of Teeth	Pitch Diameter	Outside Diameter	Bottom Diameter
	Inches	Inches	Inches
6 · 7 8 9 10 11 12 13 14 15	1.935	2.260	1.610
	2.250	2.575	1.925
	2.566	2.891	2.241
	2.882	3.207	2.557
	3.198	3.523	2.873
	3.515	3.840	3.190
	3.832	4.157	3.507
	4.149	4.474	3.824
	4.466	4.791	4.141
	4.784	5.109	4.459
16 17 18 19 20 21 22 23 24 25 26 27 28 29	5.102 5.420 5.738 6.056 6.374 6.692 7.010 7.328 7.646 7.964 8.282 8.600 8.918 9.237 9.556	5.427 5.745 6.063 6.381 6.699 7.017 7.335 7.053 7.971 8.289 8.607 8.925 9.243 9.562 9.881	4.777 5.095 5.413 5.731 6.049 6.367 6.685 7.003 7.321 7.639 7.957 8.275 8.593 8.912 9.231

Calculating Diameters of Sprocket Wheels for Roller Chains

N = Number of Teeth in Sprocket P = Pitch of Chain

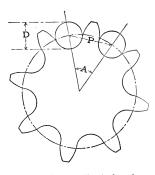
D = Diameter of Roller

__ <u>3</u>60°

Pitch Diameter = $\frac{P}{\sin \frac{1}{2} A}$

Outside Diameter = Pitch Diameter + DBottom Diameter = Pitch Diameter - D

Fig. 10



Diameter of Sprocket Wheels for Roller Chains I" pitch where D = .45.

DIAMETER OF SPROCKET WHEELS FOR ROLLER CHAINS OF I" PITCH WHEN D = .45''

No. of Teeth	Pitch Diameter in Inches	Outside Diameter in Inches	Bottom Diameter in Inches
6 7 8 9 10 11 12 13 14 15 16 17	2. 2.305 2.613 2.923 3.236 3.549 3.863 4.179 4.494 4.809 5.125 5.442 5.758 6.122	2.45 2.755 3.063 3.373 3.686 3.999 4.313 4.629 4.944 5.259 5.575 5.892 6.208 6.572	1.55 1.855 2.163 2.473 2.786 3.099 3.413 3.729 4.044 4.359 4.675 4.992 5.308 5.672
20 21 22 23 24 25 26 27 28 29	6.392 6.747 7.025 7.344 7.661 7.979 8.296 8.614 8.932 9.249 9.566	6.842 7.197 7.475 7.794 8.111 8.429 8.746 9.064 9.382 9.699	5.072 5.942 6.297 6.575 6.894 7.211 7.529 7.846 8.164 8.482 8.799 9.116

A TABLE FOR DIMENSIONS FOR MITER GEARS

The accompanying table is of service in determining the principal dimensions of miter gears (center angle 45 degrees), the number of teeth and the pitch being known. The table covers most of the possible number of teeth from 12 to 60, inclusive, and pitches from 2 to 10, inclusive, omitting 9. Values for face and cut angles correspond with designations in Fig. 11.

_			Pitch Diameters	Diar	neter	S		=:			1	ide J	Jiameters				Face	Ü
No. Tee	P. 3	P. 4	P. 5	P. 6	P. 7	P.'8	- E-	.dcı	2 P.	3 P.	4 P.	5 P.	6 P.	7 P.	8 P.	10 P.	Angle	Angle
<u> </u>	10	4	3	0900	10	102	-t:	110	6.71	4.48	3.35	2.68	2.24	1.92	1.68	1.34	51,43,	37,14
_	- 2	45°	34.	esse.	200	1 %	disc.	OII	7.21	4.80	3.00	2.88	2.40	2.00	1.81	1.44	51,12	37,4
	7	45	-/21	-90	273	CI.	H 214	1,0	7.71	5.14	3.85	3.08	2.57	2.20	1.93	1.54	50,47	38.2
	73	2	rol-ve	~	200	-22-	- 19 H	J.	8.21	5.46	4.10	3.28	2.73	2.35	2.06	1.64	50,23	38,4
_	000	-(c.	4	31	200	- C)	61	Igo	8.71	5.80	4.35	3.48	2.90	2.49	2.18	1.74	50,03	39,0
~	-60	No.	44	:45	23	53	2,2	110	0.21	6.14	4.60	3.08	3.07	2.03	2.31	1.84	49,45	39,3
_		9	4	200		24	27	E L	0.71	6.48	4.85	3.88	3.24	2.77	2.43	1.04	49,29,	39 4
_	10	63	E 4	33	3.5	22	(N)	011	10.21	6.80	5.10	4.08	3.40	2.03	2.50	2.04	49 IS	40.0
ĭ	. 0	63	7		04.5	- X-	- C1	61	10.71	7.14	5.35	4.28	3.57	3.06	2.08	2.14	49 03	40°1
ĭ	**0	1	5.1	-0.	roks	3	(A)	2 1	11.21	7.46	5.60	4.48	3.73	3.20	2.81	2.24	48,51	40°3
н	' H	73	5.5	CVA.C	3.4	33	24	21.0	11.71	7.80	5.85	4.68	3.00	3.35	2.03	2.34	48'41'	4004
н	£1.3	240	, rc	Sec.	200	333	1-100	210	12.21	8.14	01.0	4.88	4.07	3.40	3.06	2.44	48'31'	40,5
H			9	430	4	333	2	5 1	12.71	8.48	6.35	2.08	4.24	3.63	3.18	2.54	48,22,	41.0
H	-40	83	64		4	3.7	~~	2 2	13.21	.8.80	09.9	5.28	4.40	3.77	3.31	2.04	48°14′	4101
H	_	825	0.5	-0.5	543	3,5	3	200	13.71	9.14	6.85	5.48	4.57	3.02	3.43	2.74	48,07,	41,2
H	100	0	63	0.00	24	343	200	2	14.21	9.40	7.10	5.08	4.73	4.00	3.50	2.84	48,00,	41,3
H	4	10	_	min.	4.0	4	- C)	z o	14.71	9.80	7.35	5.88	8.9	4.20	3.68	2.04	47,54	41,3
H	-400	93	-4+	55	466	45	300	210	I5.2I	10.14	2.60	6.08	5.07	4.35	3.81	3.04	47,47	41,4
н	2	0	-/	9	S.		ري اينينا	3	15.71	10.48	7.85	6.28	5.24	4-49	3.93	3.14	47,42	41,5
н	53 I	103	7	19	58	4 س	38.	310	16.21	10.80	8.10	6.48	5.40	4.63	4.06	3.24	47,37	41,5
Ĥ	_	0		920	50%	434	4	310	16.71	11.14	8.35	89.9	5.57	4.77	4.18	3.34	47,32	420
Ĥ	16½ I	н	-4-0	ene 600	250	45	-4s	310	17.21	11.46	8.60	6.88	5.73	4.03	4.31	3.44	47,27,	42,10
_	17 1	13		450	500	4 4 4	4	310	17.71	11.80	8.85	2.08	2.00	5.00	4.43	3.54	47,23	42 1
-	_	nto H	×	_		25	28.8	310	18.21	12.14	01.6	7.28	0.07	2.30	4.50	3.04	47,19	42,1
H		61	6	7.00	9	53	45.	310	18.71	12.48	9.35	7.48	6.24	5.35	4.68	3.74	47,15	42,2
H	183 I	23	9	2100	00	5%	2000	310	19.21	12.80	09.6	2.68	6.40	5.40	4.81	3.8.1	47,11	42,2
Ĥ	I 6:	(N)	20	720	63	- Ye	4	310	16.71	13.14	9.85	7.88	6.57	5.03	4.93	3.04	47,08,	42,3
٦	00	33	0	20	190	55	S	4	20.71	13.80	10.35	8.28	0.90	5.03	5.18	4.14	47 or	42,3
8		4	203	80	_	9	237	4 10	21.71	14.48	10.85	89.8	7.24	6.20	5.43	4.34	46°56′	42°4
61	1 2		-	200	789	63	-C2	414	22.71	15.14	11.35	9.08	7.57	6.49	2.08	4.54	46°50′	42,2
8	3 1	153	113	0 10	7,	- €9		410	23.71	15.80	11.85	9.48	2.00	6.77	5.93	4.74	46°46′	42°5
61	4 1	9	2	983	∞	67	9	410	24.71	16.48	12.35	9.88	8.24	2.00	6.18	4.04	46,42	43.0
(1)	25		-60		80	1	6	25	25.71	17.14	12.85	10.28	8.57	7.35	6.43	5.14	46,37,	43°I
61	1 2	∞	(3½ IO	108	6	40	9	510	27.71	18.48	13.85	11.08	9.24	7.92	6.93	5.54	46,31	43°I
Ø	0	103	[42] I	D10	9	200	7	512	29.71	19.80	14.85	11.88	0.60	8.40	7.43	5.04	46,24	43,2
_	-			,														

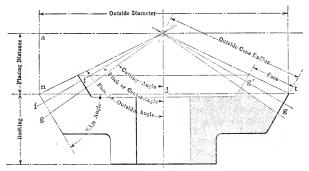


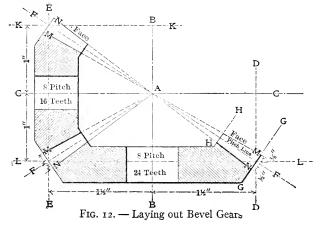
Fig. 11. — Bevel Gear Parts

BEVEL GEARS

Bevel Gears are used to transmit power when shafts are not parallel. They can be made for any angle, but are more often at right angles than any other. Right angle bevel gears are often called miter gears. The teeth are or should be radial so that they are longer at the outer end. The pames of parts are shown in Fig. 11. These should be noted carefully, particularly the face angles. The earlier editions measured face angle at right angles to the axis, but this is now changed as shown.

LAYING OUT BEVEL GEAR BLANKS

In laying out bevel gears, first decide upon the pitch, and draw the center lines B B and C C, intersecting at right angles at A as shown in Fig. 12. Then draw the lines D D to E E the same distance each



side of B B and parallel to it; the distance from D D to E E being as many eighths of an inch—if it be 8 pitch—as there are to be teeth in the gear. In the example the number of teeth is 24; therefore the distance from D D to E E will be $\frac{24}{8}$, or $1\frac{1}{2}$ inches each side of B B. K K and L L are similarly drawn, but there being only 16 teeth in the small gear, the distance from K K to L L will be $\frac{16}{8}$, or 1 inch each side of C C. Then through the intersections of D D and L L, E E and L L, and E E and K K, draw the diagonals F A. These are the pitch lines. Through the same point draw lines as G G at right angles to the pitch lines, forming the backs of the teeth. On these lines lay off $\frac{1}{8}$ of an inch each side of the pitch lines, and draw M A and N A, forming the faces and bottoms of the teeth. The lines H H are drawn parallel to G G, the distance between them being the width of the face.

The face of the larger gear should be turned to the lines M A, and the small gear to N A. For other pitches the same rules apply. If 4 pitch, use 4ths instead of 8ths; if 3 pitch, 3ds, and so on.

Bevel gears should always be turned to the exact diameters and angles of the drawings and the teeth cut at the correct angle.

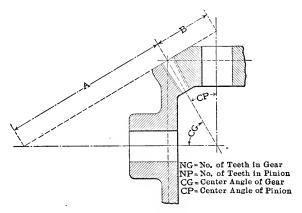


Fig. 13. — Finding the Cutter to Use

Proportions of Miter and Bevel Gears

To Find the Pitch or Center Angle:

Divide the number of teeth in the gear by the number of teeth in the pinion. This gives the tangent of the pitch angle of the gear. Or divide the number of teeth in the pinion by the teeth in the gear and get the tangent of the pitch angle of the pinion. Subtracting either pitch angle from 90 gives the pitch angle of the other.

To Find the Outside Diameter:

Multiply the cosine of the pitch angle by twice the addendum and add the pitch diameter.

To Find the Outside Cone Radius or Apex Distance:

Multiply the secant of the pitch angle of the pinion by $\frac{1}{2}$ the pitch diameter of the gear.

To Find the Face and Cutting Angles:

Divide the addendum by the outside cone radius or apex distance. This gives the tangent of the addendum or outside angle. Subtract this angle from the pitch angle of the pinion to obtain the cutting angle of the pinion, and the face angle of the gear. Subtract the same addendum angle from the center angle of the gear to obtain the cutting angle of the gear and the face angle of the pinion. This gives a uniform clearance and is especially for use with rotary cutters.

To Find Hight of Addendum at Small End of Tooth:

Divide the addendum at the large end of the tooth by the outside cone radius. This gives the decrease in hight of the addendum for each inch of gear face. Multiply this by the length of the gear face and subtract the result from the addendum of the large end of the tooth. The difference is the hight of the addendum at the small end of the tooth.

CUTTERS FOR BEVEL GEARS

Lay out the bevel gears and draw lines A and B at right angles to the center angle line. Extend this to the center lines and measure A and B. The distance A = the radius of a spur gear of the same pitch, and finding the number of teeth in such a gear we have the right cutter for the bevel gear in question. Calling the gears 8 pitch and the distance A = 4 inches. Then $2 \times 4 \times 8 = 64$ teeth, so that a No. 2 cutter is the one to use. For the pinion, if B is 2 inches, then $2 \times 2 \times 8 = 32$ or a No. 4 cutter is the one to use.

USING THE BEVEL GEAR TABLE

Take a pair of bevel gears 24 and 72 teeth, 8 diametral pitch. Divide the pinion by the gear $-24 \div 72 = .3333$. This is the tangent of the center angle of pinion. Look in the seven columns under center angles for the nearest number to this. The nearest is .3346 in the center column, as all these are decimals to four places. Follow this out to the left and find 18 in the center angle column. As the .3346 is in the column marked .50 the center angle of the pinion is 18.50 degrees. Looking to the right under center angles for gears find 71 and add the .50 making the gear angle 71.50 degrees. Thus:

Center angle of pinion 18.5 degrees. Center angle of gear 71.5 degrees.

In the first column opposite 18 is 36. Divide this by the number of teeth in the pinion, 24, and get 1.5 degrees. This is the angle increase for this pair of gears, and is the amount to be added to the center angle to get the face angle and to be deducted to get the cut angles. This gives

Pinion center angle 18.5 + 1.5 = 20 degrees face angle. Pinion center angle 18.5 - 1.5 = 17 degrees cut angle. Gear center angle 71.5 + 1.5 = 73 degrees face angle. Gear center angle 71.5 - 1.5 = 70 degrees cut angle.

For the outside diameter go to the column of diameter increase and in line with 18 find 1.90. Divide this by the pitch, 8, and get .237, which is the diameter increase for the pinion. Follow the same line to the right and find .65 for the gear increase. Divide this by the pitch, .8, and get .081 for gear increase. This gives

Pinion, 24 teeth, 8 pitch = 3 inches + .237 = 3.237 in. outside dia. Gear, 72 teeth, 8 pitch = 9 inches + .081 = 9.081 in. outside dia.

To Select the Cutter

Another way of selecting the cutter is to divide the number of teeth in the gear by the cosine of the center angle C and the answer is the number of teeth in a spur gear from which to select the cutter. For the pinion the process is the same except the number of teeth in the pinion is divided by the sine of the center angle. Formula

Tangent of
$$CG = \frac{NG}{NP}$$
. Tangent of $CP = \frac{NP}{NG}$.

Number of teeth to use in selecting cutter for gear $= \frac{NG}{Cos\ CG}$.

Number of teeth to use in selecting cutter for pinion = $\frac{NP}{Sin\ CG}$

Any pair of gears can be figured out in the same way, bearing in mind that when finding the center angle for the gear, to read the parts of a degree from the decimals at the bottom, and that for the pinion they are at the top. In the example worked out the tangent came in the center column so that it made no difference. If, however, the tangent had been .3476 we read the pinion angle at the top, 19.17 degrees and the gear angle at the bottom, 70.83. By noting that the sum of the two angles is 90 degrees, we can be sure we are right.

BEVEL GEAR TABLE SHAFT ANGLES 90°

Teeth	r In- ide by finion	ngle s for		Cente	r Angle	Hundi	redth D	egrees		ngle r Gear	r In- ide by
Amgle Increase; Divide by Teeth in Pinion	Diameter In- crease; Divide by Pitch for Pinion	Center Angle Degrees for Pinion		Lef	t-hand	Colum	n read l	here		Center Angle Degrees for Gear	Diameter In- crease; Divide by Pitch for Gear
Amg Div	Dia cres Pitc	Cen	0	.17	•33	.50	.67	.83	1.00	Ce n Deg	Crea Pite
2 4 6 8 10 12 14 16 18 20 22 24	2.00 2.00 2.00 1.99 1.99 1.98 1.98 1.98 1.97 1.96	1 2 3 4 5 6 7 8 9 10 11 12	.0175 .0349 .0524 .0699 .0875 .1051 .1228 .1405 .1584 .1763 .1944	.0204 .0378 .0553 .0729 .0904 .1080 .1257 .1435 .1614 .1793 .1974	.0233 .0407 .0582 .0758 .0934 .1110 .1278 .1465 .1644 .1823 .2004	.0262 .0437 .0612 .0787 .0963 .1139 .1317 .1495 .1673 .1853	.0291 .0466 .0641 .0816 .0992 .1169 .1346 .1524 .1703 .1883	.0320 .0495 .0670 .0846 .1022 .1198 .1376 .1554 .1733 .1914 .2095	.0349 .0524 .0699 .0875 .1051 .1228 .1405 .1584 .1763 .1944 .2126	89 88 87 86 85 84 83 82 80 79 78	.07 .10 .14 .17 .21 .24 .28 .31 .34 .38 .41
26 28 30 32 34 36 37	1.95 1.94 1.93 1.92 1.91 1.90 1.89	13 14 15 16 17 18 19	.2309 .2493 .2679 .2867 .3057 .3249 .3443 .3640	.2339 .2524 .2711 .2899 .3089 .3281 .3476 .3673	.2370 .2555 .2742 .2931 .3121 .3508 .3706	.2401 .2586 .2773 .2962 .3153 .3346 .3541 .3739	.2432 .2617 .2805 .2994 .3185 .3378 .3574 .3772	.2462 .2648 .2836 .3026 .3217 .3411 .3607	.2493 .2679 .2867 .3057 .3249 .3443 .3640 .3839	76 75 74 73 72 71 70 69	.45 .48 .51 .55 .58 .62 .65 .68
41 43 45 47 49 50 52 54	1.86 1.85 1.84 1.82 1.81 1.79 1.78 1.76	21 22 23 24 25 26 27 28	.3839 .4040 .4245 .4452 .4663 .4877 .5095	.3872 .4074 .4279 .4487 .4699 .4913 .5132 .5354	.3906 .4108 .4314 .4522 .4734 .4950 .5169	.3939 .4142 .4348 .4557 .4779 .4986 .5206	.3973 .4176 .4383 .4592 .4806 .5022 .5243 .5467	.4006 .4210 .4417 .4628 .4841 .5059 .5280	.4040 .4245 .4452 .4663 .4877 .5095 .5317	68 67 66 65 64 63 62 61	.75 .78 .81 .84 .88 .91 .93
56 57 59 61 63 64 66 68	1.74 1.73 1.71 1.69 1.67 1.65 1.63	29 30 31 32 33 34 35 36	.5543 .5774 .6009 .6249 .6494 .6745 .7002	.5581 .5812 .6048 .6289 .6536 .6787	.5619 .5851 .6088 .6330 .6577 .6830	.5658 .5890 .6128 .6371 .6619 .6873	.5696 .5930 .6168 .6412 .6661 .6916	.5735 .5969 .6208 .6453 .6703 .6959	.5774 .6009 .6249 .6494 .6745 .7002	59 58 57 56 55 54	1.00 1.03 1.05 1.08 1.11 1.14
69 71 72 73 75 77 79 80 81	1.59 1.57 1.55 1.53 1.51 1.48 1.46 1.43	37 38 39 40 41 42 43 44 45	.7203 .7536 .7813 .8098 .8391 .8693 .9004 .9325 .9657	.7310 .7581 .7860 .8146 .8441 .8744 .9057 .9380 .9713	.7355 .7627 .7907 .8195 .8491 .8796 .9110 .9435 .9770	.7400 .7673 .7954 .8243 .8541 .8847 .9163 .9490 .9827	.7445 .7720 .8002 .8292 .8591 .8899 .9217 .9545 .9884	.7490 .7766 .8050 .8342 .8642 .8952 .9271 .9601 .9942	.7536 .7813 .8098 .8391 .8693 .9004 .9325 .9657 1.0000	53 52 51 50 49 48 47 46 45 44	1.20 1.23 1.25 1.28 1.31 1.33 1.36 1.39 1.41
1		1 70	1.00	.83	.67	•50	-33	.17	•		1 - 40

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126 GEARING

SPIRAL GEARS

THE term spiral gear is usually applied to gears having angular teeth and which do not have their shafts or axis in parallel lines, and usually at right angles. Spiral gears take the place of bevel gears and give a smoother action as well as allowing greater speed ratios in a given space. When gears with angular or skew teeth run on parallel shafts they are usually called helical gears.

THE CALCULATION OF FORTY-FIVE DEGREE SPIRAL GEARS

Pitch of Cut- ter	Pitch Diam.	Pitch of Spiral in Inches to One Turn	No. of Teeth in Spur Same Curva- ture	Outside Diam.	Thick- ness of Tooth at Pitch Line (Normal)	Depth of Tooth	Clear- ance	Circular Pitch (Normal)
		y by Num in Spiral (Add to P.D.				
2 2 1 2 1 2 1 2 2 3 3	0.70710 0.62855 0.56566 0.51425 0.47140	2.22142 1.97464 1.77707 1.61556 1.48094	2.828 2.828 2.828 2.828 2.828 2.828	1.0000 0.8888 0.8000 0.7273 0.6666	0.7854 0.6981 0.6283 0.5712 0.5236	1.0785 0.9587 0.8628 0.7844 0.7190	0.0785 0.0699 0.0628 0.0572 0.0524	1.5708 1.3963 1.2566 1.1424 1.0472
3 ¹ / ₂ 4 5 6 7	0.40406 0.35355 0.28283 0.23570 0.20203	1.26939 1.11071 0.88853 0.74047 0.63469	2.828 2.828 2.828 2.828 2.828 2.828	0.5714 0.5000 0.4000 0.3333 0.2857	0.4488 0.3927 0.3142 0.2618 0.2244	0.6163 0.5393 0.4314 0.3595 0.3081	0.0449 0.0393 0.0314 0.0262 0.0224	0.8976 0.7854 0.6283 0.5236 0.4488
8 9 10 11 12	0.17677 0.15714 0.14143 0.12856 0.11785	0.55534 0.49367 0.41431 0.40388 0.37024	2.828 2.828 2.828 2.828 2.828 2.828	0.2500 0.2222 0.2000 0.1818 0.1666	0.1963 0.1745 0.1571 0.1428 0.1309	0.2696 0.2397 0.2157 0.1961 0.1798	0.0196 0.0175 0.0157 0.0143 0.0131	0.3927 0.3491 0.3142 0.2856 0.2618
14 16 18 20 22	0.10101 0.08836 0.07855 0.07071 0.06428	0.31733 0.27759 0.24677 0.22214 0.20194	2.828 2.828 2.828 2.828 2.828 2.828	0.1429 0.1250 0.1111 0.1000 0.0909	0.1122 0.0982 0.0873 0.0785 0.0714	0.1541 0.1348 0.1198 0.1079 0.0980	0.0112 0.0098 0.0088 0.0079 0.0071	0.2244 0.1963 0.1745 0.1571 0.1428
24 26 28 30 32	0.05892 0.05437 0.05050 0.04713 0.04425	0.18510 0.17081 0.15865 0.14806 0.13901	2.828 2.828 2.828 2.828 2.828 2.828	0.0833 0.0769 0.0714 0.0666 0.0625	0.0654 0.0604 0.0561 0.0524 0.0491	0.0898 0.0829 0.0770 0.0719 0.0674	o.oo65 o.oo60 o.oo56 o.oo53 o.oo50	0.1309 0.1208 0.1122 0.1047 0.0982
36 40 48	0.03929 0.03533 0.02944	0.12343 0.11099 0.09249	2.828 2.828 2.828	0.0555 0.0500 0.0417	0.0436 0.0393 0.0327	0.0599 0.0539 0.0449	0.0043 0.0039 0.0033	0.0785 0.0873 0.0654

In considering speed ratios for spiral gears the driving gear can be taken as a worm having as many threads as there are teeth and the driven as the worm wheel with its number of teeth, so that one revolution of the driver will turn a point on the pitch circle of the driven gear as many inches as the lead of the teeth of the driver. Divide this by the circumference of the pitch circle of the driven gear

to get the revolutions of the driven.

While the subject of spiral gears is rather complex if considered broadly, most of the difficulties disappear when they have a tooth angle of 45 degrees. It is perhaps for this reason that from 75 to 90 per cent. of the spiral gears used are made with this angle.

This has the added advantage of being the most durable, although there is but a trifling increase in wear down to 30 degrees and the wear at 20 degrees is not serious. In cases of necessity even 12 degrees

can be used without destructive wear.

Where higher speed ratios than can be had with a 45-degree angle tooth are necessary, they can be laid out as will be shown later and can be cut on most milling machines. The usual change gears allow about two thousand different spirals to be cut.

Where the angles are not 45 degrees, the gear with the greatest

angle must always be the driver.

All of the tooth parts are derived from the normal pitch while the pitch diameters are derived from the circular pitch. These are never the same in two gears of a pair except when both are 45 degrees.

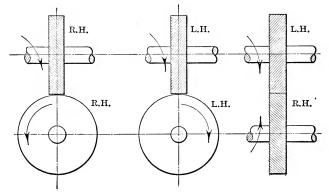
As the diameter of a spiral gear does not indicate its speed ratio, the terms driven and follower are used in place of gear and pinion.

45-DEGREE SPIRAL GEAR

These gears are the simplest of all spirals to lay out and to make, the required speed ratios being obtained by varying the diameters, precisely as with spur or bevel gears, the rules for the speed ratio being the same in both cases. Moreover, the various factors required in laying out and making such gears can be reduced to the simple table shown.

Such a table has been worked out by E. J. Kearney. With it any one can quickly make the few calculations connected with any pair of 45-degree gears having teeth between 2 and 48 diametral pitch.

This table will be found on preceding page:



EXAMPLE

Let it be desired to construct a pair of spiral gears with 35 teeth in the gear and 16 teeth in the pinion, using a 10 pitch cutter. Using table on page 126 we have

Pitch diameter = $0.14143 \times 35 = 4.950$.

Outside diameter = 4.950 + 0.200 = 5.150.

Pitch in inches to one turn of spiral = $0.44431 \times 35 = 15.550$.

NOTE.—A slight variation in one turn makes no practical difference, hence the ordinary change gears furnished with a universal miller will usually be found sufficient.

Number of teeth in spur with same curvature = $2.828 \times 35 =$

98.980.

Looking at B & S spur-gear cutter list, we see that 99 is between

55 and 134, therefore we select a No. 2 cutter.

In a similar manner using 16 as a multiplier we obtain the data for the pinion. This gives 2.262 as pitch diameter so that the center distance $= \frac{4.950 + 2.262}{2} = 3.606$.

THE VARIOUS DIMENSIONS FOLLOW

	Gear	Pinion
Number of teeth Pitch diameter Outside diameter Pitch in inches to one turn Angle of spiral Pitch of cutter No. of cutter Whole depth of tooth Angle of shafts Center distance of shafts	35 4.950 5.150 15.550 45 10 2 0.216 90 3.6	

FIGURING SPIRAL GEARS

As there is no direct solution for a pair of spiral gears their calculation is a tedious process and the result must be found by trial.

As numerous calculations are absolutely necessary, this formula should not involve division by large or fractional numbers and should contain the fewest possible operations. Such formulas are:

Let

C =Center distance,

P = Diametral pitch,

 $N_1 =$ Number of teeth in the driver,

 N_2 = Number of teeth in the follower,

 $S_1 =$ Spiral angle of driver,

 S_2 = Spiral angle of follower.

Then,

$${}_{2}C = \frac{(secant \ S_1 \ N_1) + (secant \ S_2 \ N_2)}{P}.$$

That is, the sum of the secant of the driving angle times the number of teeth in the driver, and the secant of the follower angle times the number of teeth in it divided by the diametral pitch equals twice the center distance. This formula is derived as follows: The secant of the spiral angle times the pitch diameter of a spur gear of the same number of teeth and pitch equals the pitch diameter of a spiral gear of that angle, the pitch of the spur gear being the same as the normal pitch of the spiral gear. Now for a spur gear the number of teeth divided by the diametral pitch equals the pitch diameter. Therefore,

the secant of the spiral angle $imes rac{N}{P} =$ the pitch diameter of a spiral gear.

The combined pitch diameters times the center distance are equal to

$$\left(Secant S_1 \times \frac{N_1}{P_-}\right) + \left(Secant S_2 \times \frac{N_2}{P_-}\right)$$

or (secant $S_1 N_1$) + (secant $S_2 N_2$) for one diametral pitch.

The quantity secant S_1 N_1 is the pitch diameter for the driver and secant S_2 N_2 is the pitch diameter of the follower. To obtain the center distance for any other pitch, it is simply necessary to divide this last result by that pitch.

A table of secants will furnish constants covering the entire range of angles; and therefore, all possible solutions for a pair of gears. After long experience in calculating spiral gears these are recommended by C. H. Logue as the best and simplest for all cases.

Points to be Kept in Mind when Calculating Spiral Gears

To assist in their use the following points should be kept in mind:

- 1. The diameter of a spiral gear increases with its angle.
- 2. Therefore, the diameter of the follower will reduce as the driving angle is increased, although not necessarily in the same ratio.
- 3. It is quite possible for the center distance to remain practically constant through quite a range of angles, the follower decreasing as the driver is increased. This is especially true when the gear having the greater number of teeth is the driver.

- 4. If the center distance is too great when the driving angle is 45 degrees it must not go below that in any case —a lower number of teeth must be selected for both driver and follower, while maintaining the same ratio, and another trial made using a much higher angle for the driver.
- 5. The center distance will increase with the angle of the driver. This increase is more rapid when reducing than when increasing the speed of the follower.
- 6. The number of teeth selected for each trial must be in proportion to the desired ratio.
- 7. Forty-five degrees is commonly accepted as the most efficient driving angle.

Selecting Secants and Trial Numbers of Teeth

To calculate a pair of spiral gears, select secants for the desired angles, assuming the normal pitch, try out the value of 2 C with trial numbers of teeth for driver and follower.

If the value 2 C is too small increase the number of teeth and try again. A very few calculations will show the number of teeth to

secure the closest result.

If the center distance thus found is not as desired the angles must be shifted, keeping in mind the general laws governing the change of the center distance with the angle.

It is often found that when the desired center distance is reached the driving angle is too large to be desirable. The only alternative is to change the normal pitch and try again. A slide rule will give

approximate results.

When there are limitations placed on the diameter of one or both of the gears the following formula is of value. It may also serve as a check on the above calculations. The pitch diameters are assumed.

$$Tan. \ S_1 = \frac{\left(\begin{array}{c} pitch \ diameter \ of \ driver \times \\ number \ of \ revolutions \ of \ driver \end{array} \right)}{\left(\begin{array}{c} pitch \ diameter \ of \ follower \times \\ number \ of \ revolutions \ of \ follower \end{array} \right)}.$$

This will set a limit on the driving angle S₁, to exceed which means that the gear will be too large.

REAL PITCHES FOR CIRCULAR PITCH SPIRAL GEARS

THE accompanying table will be found convenient in figuring particulars for spiral gearing, as it eliminates much of the work by shortening the process, thus making it quite an easy and simple matter to find the dimensions for either helical gears with axes parallel to each other or for gears with right-angle drive.

Formulas for use with the table are as follows: Circumference on

pitch line = real pitch multiplied by number of teeth.

Lead of Spiral = Circumference on pitch line divided by the tangent.

Pitch Diameter = Circumference divided by 3.1416.

For whole diameter add the same amount above pitch line as for

spur wheels of the same pitch as the normal pitch.

The following is an example of the use of the table: A pair of wheels is required to be: Ratio, 6 to 1; normal pitch, 1 in.; driver, 6 teeth; follower, 36 teeth; angle for driver, 66 degrees; angle for follower, 24 degrees.

Referring to the table we find that the real pitch for the driver is 2.4585.

 2.4585×6 (teeth) = 14.751 (circumference on pitch line).

 $Cir. 14.751 \div 2.246 (tangent) = 6.567 (lead of spiral).$

Cir. $14.751 \div 3.1416 = 4.695$ (pitch diameter).

For the follower the real pitch is 1.0946.

 $1.0946 \times 36 = 39.4056$ (circumference).

Cir. 39.4056 \div 0.4452 (tangent) = 88.512 (lead of spiral).

Cir. $39.4056 \div 3.1416 = 12.543$ (pitch diameter).

Another method of finding the lead of spiral is to multiply the real pitch by the number of teeth, but for this purpose take the real pitch of the mating wheel.

In the above example we should have

Real pitch of follower, 1.0946 \times 6 = 6.5676 (lead of spiral).

Real pitch of driver, $2.4585 \times 36 = 88.506$.

It will be noticed that there is a slight difference in the result but this is unimportant, as it is only brought about by the dropping of a few decimal points in the tangent. 132

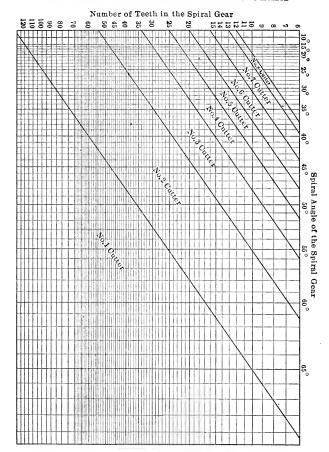
GEA	
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Axis Driver Fol- 80° 10°er	_													
		1,,,	e T	16"	en 20	3,1	1,2	1. " I	A101	2,4		8 1	614	3//
	Driver	Fol- lower	Driver	Fol- lower	Driver	Fol- lower	Driver	Fol- lower	Driver	Fol- lower	Driver	Fol- lower	Driver	Fol- lower
	1.4306	0.2538	1.7006	0.3173	2.1505	0.3808	2.5194	0.44.12	2.8793	0.5077	3.5992	0.6346	4.3190	0.7616
70 II	1.3102	0.2546	1.6377	0.3183	1.0653	0.3820	2.2028	0.4457	2.6304	0.5093	3.2755	0.6366	3.9306	0.7640
78° 12°	1.2024	0.2556	I.5030	0.3105	1.8036	0.3833	2.1012	0.4473	2.40.19	0.5112	3.0000	0.6390	3.6072	0.7666
77° 13°	I.III.3	0.2565	1.3892	0.3207	1.6669	0.3848	. I.9449	0.4490	2.2226	0.5131	2.7784	0.6414	3.3338	0.7696
76° 14°	1.0334	0.2576	1.2017	0.3220	1.5500	0.3865	I.8084	0.4500	2.0668	0.5153	2.5834	0.6.140	3.1000	0.7730
75° 15°	0.0650	0.2501	I.2074	0.3238	1.4.189	0.3885	1.6903	0.4529	1.9318	0.5181	2.4148	0.6476	2.8978	0.777
74° 16°	0.0000	0.2600	1.1337	0.3250	1.3605	0.390I	I.5872	0.4551	1.8139	0.5201	2.2674	0.6502	2.7210	0.7802
73° 17°	0.8550	0.2614	1.0688	0.3268	1.2826	0.3931	I.4964	0.4575	1.7101	0.5228	2.1377	0.6536	2.5652	0.7842
72° I8°	0.8000	0.2628	I.OII2	0.3286	1.2135	0.3943	I.4157	0001:0	1.6180	0.5257	2.0225	0.6572	2.4270	0.788
710 10	0.7678	0.2644	0.0508	0.3305	1.1518	0.3966	I.3438	0.4627	I.5357	0.5288	9616.1	0.6610	2.3036	0.793
70° 20°	0.7300	0.2660	0.9137	0.3325	1.0964	0.3000	1.2791	0.4656	1.4619	0.5320	1.8274	0.6651	2.1928	0.708
60° 21°	0.6976	0.2678	0.8720	0.3347	1.046.1	7101.0	1.2208	989t-o	1.3952	0.5356	I.7.140	0.6694	2.0028	0.803
68° 22°	0.6673	0.2696	0.8342	0.3370	I.ooio	0.1011	1.1679	0.4718	1.33.16	0.5392	1.668.4	0.6740	2,0020	0.8088
67° 23°	0.6398	0.2716	0.7998	0.3395	0.9597	0.407.4	1.1120	0.4752	1.2796	0.5432	I.5996	0629.0	1.9194	0.8148
66° 24°	9+19.0	0.2736	0.7683	0.3.120	0.0220	0.4105	1.0756	0.4789	1.2202	0.5472	1.5366	0.6841	1.8440	0.8210
65. 25.	0.5915	0.2758	0.7394	0.3448	0.8873	0.4137	1.0352	0.4827	1.1830	0.5516	1.4788	0.6896	I.7746	0.8274
64° 26°	0.5702	0.2781	0.7129	0.3477	0.8554	0.4172	0.0980	0.4868	1.1406	0.5563	1.4258	0.6054	1.7108	0.8344
63 27	0.5507	0.2806	0.6883	0.3507	0.8200	0.4200	0.9636	0.4010	1.1014	0.5612	1.3766	0.7014	I.6520	0.8418
62 28	0.5325	0.2831	0.6656	0.3539	0.7988	0.4247	0.9319	0.4955	1.0650	0.5662	1.3312	0.7078	I.5976	0.8494
61, 29,	0.5157	0.2858	0.6446	0.3573	0.7735	0.4287	0.9024	0.5002	1.0314	0.5716	I.2892	0.7146	I.5470	0.8574
	0.5000	0.2886	0.6250	0.3608	0.7500	0.4330	0.8750	0.5051	I.0000	0.5773	1.2500	0.7216	1.5000	0.8660
59, 31,	0.4853	0.2916	0.6067	0.3646	0.7281	0.4375	0.8494	0.5104	0.9708	0.5833	1.2134	0.7292	1.4502	0.8750
58, 32,	0.4717	0.2048	0.5897	0.3685	0.7076	0.4422	0.8256	0.5159	0.9435	0.5890	1.1794	0.7370	1.4152	0.8844
57, 33,	0.4200	0.2081	0.5738	0.3726	0.6885	0.4471	0.8033	0.5216	0.0180	0.5962	1.1476	0.7452	1.3770	0.8942
56 34	0.4470	0.3015	0.5588	0.3769	0.6706	0.4523	0.7824	0.5277	0.8941	0.6031	1.1176	0.7538	1.3412	0.0040
55 35	0.4358		0.5448	0.3815	0.6538	0.4578	0.7627	0.5341	0.8717	0.6104	1.0896	0.7630	1.3076	0.015
54, 36,	0.4253		0.5316	0.3862	0.6380	0.4635	0.7443	0.5408	0.8506	0.6180	1.0632	0.7724	I.2760	0.9270
53. 37.	0.4154	0.3130	0.5192	0.3913	0.6231	0.4695	0.7269	0.5478	0.8308	0.6260	1.0384	0.7826	1.2462	0.0390
52 38	0.4060	0.3172	0.5076	0.3965	1609.0	0.4759	0.7161	0.5552	0.8121	0.6345	1.0152	0.7930	1.2182	0.0518
51, 39,	0.3972		0.4965	0.4021	0.5959	0.4825	0.6952	0.5628	0.7945	0.643.1	0.9930	0.8042	8161.1	0.9650
50, 40°	0.3889	_	0.4861	0.4079	0.5834	0.4895	0.6806	0.5711	0.7778	0.6527	0.9722	0.8159	1.1668	0.0790
49° 41°	0.3810	_	0.4763	0.4140	0.5716	0.4969	0.6668	0.5797	0.7621	0.6625	0.9526	0.8280	I.1432	0.9938
48° 42°	0.3736	_	0.4670	0.4205	0.5604	0.5046	0.6538	0.5887	0.7472	0.6728	0.9340	0.8410	1.1208	1.000
47 43	0,3665	0.3418	0.4582	0.4272	0.5498	0.5127	0.6415	0.5982	0.7331	0.6836	0.0164	0.8544	0660.1	1.0254
46, 44,	0.3598	0.3475	0.4498	0.4344	0.5398	0.5213	0.6298	0.6082	0.7197	0.6950	0.8996	0.8688	1.0796	1.0420

18	E	Tangent of Angle	Fol- lower	lower		I.5280	.21.46 1.5335 4.7046 0.2126	.668r 1.5394 4.3315 0.2309	4.0108	3.7321	I.5604 3.4874	I.5685 3.2709	_	1.5804	1.5962 2.7475	1.6007	1.6178 2.4751	I.6295	1.6418		4218 I.6689 2.0503 0.4877	1.0835	1.6988	:0040 I.7150 I.8040 0.5543	1.7320 I.7321	I.7499 I.6643	1.7687 1.6003	I.7885 I.5399	I.8093 I.4826	1.8311 1.4281	I.S54I I.3704	I.878I I.3270	1.9035 1.2799	I.930I I.23.49	1.9581 1.1918	2.2863 1.9875 1.1504 0.8693	2.2417 2.0184 1.1106 0.9004	3.1994 2.0510 1.0724 0.9325	2 1502 2 0852 1.0255 0.0657
Follor Driver Prol- Driver Follor Driver Prol- Driver Dri		1.3%		1	-	н	H	-	н	2	Η	_	_	н	н	_	_	_	_	_		_		-	н	H	=	н.	-	_	_		_	-	_	_		_	0702 1.0114
Fol. Driver		1,1,1	Fol- lower	TOMOT	1.2092 7	1.2732	1.2780 6	I.2828 (I.2882	I.2952 S	I.300.1	I.307I 4	I.3143 4	I.3220 4	1.3302	1.3389	1.3480	1.3580	I.3082	1.3792	I.3908		1.4157	I.4292	I.4433	1.4583	I.4740	1.4904	1.5077	_	1.5449	1.5052	1.5802	1.003.1	- ' '	(4	1.6820	I.7089 2	I.7377 I
Fol. Driver	mal Ditches		1	1		_	-	_	_	_	_				-	_		_	_	01	_			-						_	_		.4	_	-	_	_	_	_
Nationary 1.5	Nor	13"		1		_		_		_	_	_	_				_		_	_	_	_	_	_	_	_					_		_		_			_	-
Nationary 1.0		_	1			-			_	_		_	-	,	.,	_	_		_	_	_	_				_	C1 6	- N			_	_	_	_	-	_	_	_	_
Nist Driver 100 Driver 100 Driver 110		,ı	Driver	1 2 2 2	5.7507	5.2407	0600-1	4-4454	4.1330	3.8037	3.0279	3.4203	3.2300	3.0715	2.9238	2.7904	2.0094	2.5593	2.4585	2.3002	2.2511	2.2020	2.1300	2.0020	2.0000	1.9415	07501	1.0500	1.7003	1.7434	210/17	1.0014	1.0242	1.5390	1.5557	I.5240	I.4944	I.4002	I.4395
xis as from (10 of 10 of		*Iw		+	-	-					_	_	_	_	_	_	_			_	_	_	_	_				-	-	-		-	_		_	_	-	_	-
is significant with the signif			Drive	5.028	2000	× × × ×	2003	3.0000	3.0102	3.3300	3.1744	2.0922	2.0314	2.0070	2.5502	2.4410	2.3350	2.2240	2.1512	2.0704	1.9900	1.9272	1.0038	1.0040	1.7500	1.0000	1 6066	2000	1.3040	10 10 1	200	1.4550	1.4322	1.3904	1.3012	1.3330	1.3070	1.2030	1.2590
	ngles from	Axis	1	80°	_	000	0 170	6,10	47	150	01,	007	07	67	0,000	230	000	25.50	010	0,50	020	080	0000	000	o v	0,00	0 22 2	0,00	0,10	200	0,20	0000	0000	9000	24.	00.	000	45,	40, 44

134 GEARING

SPUR-GEAR CUTTERS FOR SPIRAL GEARS



To find the number of a spur-gear cutter to be used in cutting a given spiral gear, locate the intersection of lines traced from the points representing the number of teeth and the spiral angle on the two scales. The number in the area on the chart within which the intersection falls is the cutter number of Brown & Sharpe's involute cutter system required,

SPIRAL GEAR TABLE

WHILE it is better in every case to understand the principles involved before using a table as this tends to prevent errors, they can be used with good results by simply following directions carefully. The subject of spiral gears is so much more complicated than other gears that many will prefer to depend entirely on the tables.

This table gives the circular pitch and addendum or the diametral pitch and lead of spirals for one diametral pitch and with teeth having angles of from 1 to 80 degrees to 45 and 45 degrees. For other pitches divide the addendum given and the spiral number by the required pitch and multiply the results by the required number of teeth. This will give the pitch diameter and lead of spiral for each wheel. For the outside diameter add two diametral pitches as in spur gearing.

Suppose we want a pair of spiral gears with 10 and 80 degree angles. 8 diametral pitch cutter, with 16 teeth in the small gear, having 10degree angle and 10 teeth in the large gear with its 80-degree angle.

Find the 10-degree angle of spiral and in the third column find 1.0154. Divide by pitch, 8, and get .1260. Multiply this by numteeth $- .1269 \times 16 = 2.030 = pitch$ diameter. pitches – two $\frac{1}{8} = \frac{1}{4}$ and 2.030 + .25 = 2.28 inches outside diameter.

The lead of spiral for 10 degrees for small wheel is 18.002. Divide by pitch = $18.002 \div 8 = 2.2615$. Multiply by number of teeth, $2.2615 \times 16 = 36.18$, the lead of spiral, which means that it makes one turn in 36.18 inches.

For the other gear with its 80-degree angle, find the addendum, 5.7587. Divide by pitch, 8, = .7198. Multiply by number of teeth, 10 = 7.198. Add two pitches, or .25, gives 7.448 as outside

diameter.

The lead of spiral is 3.1901. Dividing by pitch, 8 = .3988. plying by number of teeth = 3.988 the lead of spiral.

When racks are to mesh with spiral gears, divide the number in

the circular pitch columns for the given angle by the required diametral pitch to get the corresponding circular pitch.

If we want to make a rack to mesh with a 40-degree spiral gear of 8 pitch: Look for circular pitch opposite 40 and find 4.101. Dividing by 8 gives .512 as the circular pitch for this angle. The greater the angle the greater the circular or linear pitch, as can be seen by trying an 80-degree angle. Here the circular pitch is 2.261 inches.

SPIRAL GEAR TABLE

SHAFT ANGLES 90° FOR ONE DIAMETRAL PITCH

Angle of Spiral Degrees	To obtain the circular pitch for one tooth, divide by the required diametral pitch	To obtain the pitch diameter, divide by the required diametral pitch and multiply the quotient by the required number of teeth	spiral, di required pitch ar quotient	the lead of ivide by the diametral ad multiply by the re- number of	To obtain the pitch diameter, divide by the required diametral pitch and multiply the quotient by the required number of teeth	To obtain the circular pitch for one tooth, divide by the required diametral pitch.	Angle of Spiral in Degrees
	Circular Pitch	One Tooth or Addend		f Spirals	One Tooth or Addendum	Circular Pitch	
Small Wheel	Small Wheel	Small Wheel	Small Wheel	Large Wheel	Large Wheel	Large Wheel	Large Wheel
		Wheel 1.0001 1.0006 1.0013 1.0024 1.0038 1.0035 1.0035 1.0098 1.0124 1.0187 1.0154 1.0187 1.0322 1.0203 1.0366 1.0352 1.0402 1.0456 1.0514 1.0771 1.0785 1.0863 1.1126 1.1233 1.1247 1.1666 1.1791 1.1023 1.12207 1.2236	Wheel 180.05 90.020 60.032 45.038 37.077 30.056 25.728 22.573 20.082 18.092 16.464 15.076 13.966 12.986 11.393 10.417 10.192 9.6494 9.1848 8.7662 8.3862 8.0399 7.7379 7.4332 7.1664 6.012 6.4790 6.2778 6.0979 5.0282 5.7710 5.6181 5.4754 5.3431 5.2201	Wheel 3.1420 3.1438 3.1492 3.1527 3.1589 3.1651 3.1671 3.2003 3.2003 3.2204 3.2278 3.2524 3.2623 3.2252 3.3253 3.34129 3.4533 3.4552 3.3433 3.4552 3.3433 3.4552 3.3433 3.4552 3.3453 3.4729 3.4952 3	Wheel 57.298 28.653 19.107 14.335 11.473 9.5667 8.2055 7.1852 6.3024 5.7587 5.2408 4.8097 4.4454 4.1335 3.8637 3.6279 3.4203 3.2360 3.0715 2.928 2.7904 2.65094 2.55593 2.4585 2.3662 2.2811 2.2026 2.2811 2.2026 2.1300 2.0620 1.0416 1.8870 1.8360 1.7882 1.7434 1.7013	Wheel 180.01 90.016 60.026 45.035 36.044 30.055 25.778 22.573 20.082 18.092 16.464 13.988 12.986 12.138 11.397 10.745 10.166 9.6494 9.1854 8.7603 8.3862 7.7242 7.4336 7.7242 7.4336 6.0107 6.0917 5.0282 5.7680 5.6178 5.4770 5.3448 5.2200	Wheel 89 87 88 87 86 85 84 80 79 76 75 77 70 60 66 66 66 66 69 55 55 56 55 56
39 40 41 42	4.0483 4.1010 4.1626 4.2273	1.2690 1.2867 1.3054 1.3250 1.3456	5.1028 4.9866 4.8873 4. 7885 4.6040	3.9921 4.0416 4.1012 4.1540 4.2272	1.6242 1.5890 1.5557 1.5242 1.4944	5.1026 4.9920 4.8874 4.7884 4.6948	52 51 50 49 48
43 44 45	4.2956 4.3671 4.4428	1.3673 1.3901 1.4142	4.6065 4.5223 4.4428	4.2272 4.2956 4.3675 4.4428	1.4662 1.4395 1.4142	4.6062 4.5225 4.4428	47 46 45

THREADS OF WORMS

Worms are cut with threads having a total angle of 29 degrees, similar to the Acme thread. Some use the same proportions as for the Acme, but most use a deeper thread such as the Brown & Sharpe, which is .6866 deep instead of .51 for a one-inch pitch as in the Acme. It is not easy to cut odd fractional pitches in most lathes, so regular pitches are cut and the circular pitch of the worm wheel is allowed to come in fractional measurements for pitch diameters and center distances. Having determined on the reduction as 40 to 1, the relative proportions can be considered as follows:

Assume a thread of 4 to the inch for the worm or a lead of $\frac{1}{4}$ inch. Then as the reduction of 40 to 1 there must be 40 teeth in the worm gear, $\frac{1}{4}$ inch from center to center of teeth or 10 inches in circumference on the pitch line or 3.18 inches. If a reduction of 20 to 1 is wanted we can use the same gear but cut a double thread of 2 per inch, which will give the same distance between teeth, but the worm

gear will be moved two teeth every revolution of the worm.

Some of the commonly used proportions are:
Pitch diam. of worm gear =
$$\frac{\text{No. of teeth} \times \text{pitch in inches}}{3.1416}$$

Diametral pitch = $\frac{3.1416}{\text{Linear pitch}}$.

Throat diam. of worm gear = Pitch diam. + $\frac{2}{\text{Diam.Pitch}^{6}}$

Outside diameter of gear for 60° sides = throat diameter + 2 (.13397 throat radius.)

Whole depth of tooth of worm or worm gear = .6866 \times linear pitch.

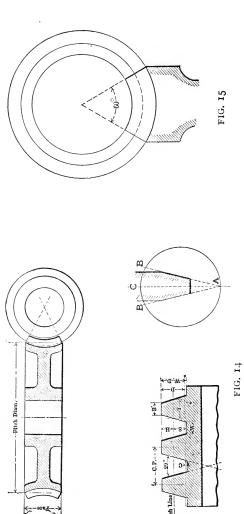
Width at top of tooth of worm = $.335 \times$ linear pitch. Width of bottom of tooth of worm = $.31 \times$ linear pitch. Outside diam. of worm – single thread = $4 \times$ linear pitch.

Outside diam. of worm – double thread = 5 × linear pitch. Outside diam. of worm – triple thread = 6 × linear pitch.

Face of worm gear $= \frac{1}{2}$ to $\frac{3}{4}$ outside diameter of worm.

Width of Face

A COMMON practice for determining the width of face or thickness of worm wheels is shown in Fig. 15. Draw the diameter of the worm and lay off 60 degrees as shown; this gives the width of working face, the sides being made straight from the bottom of the teeth. Others make the face equal to \(\frac{3}{4}\) the outside diameter of worm, but \(\frac{1}{2}\) the diameter of the worm is more common.



To FIND THE PITCH DIAMETER. (Given pitch and number of teeth.)

 $24 \div 3.1416 = 7.639 = \text{pitch}$ Rate. — Multiply the number of teeth by the circular pitch and divide by 3.1416. Example. — Number of teeth 96; Circular pitch, $\frac{1}{4}$. Then $96 \times \frac{1}{4} = 24$. $24 \div 3$.: diameter.

Note. - Diameter of Hob equals diameter of Worm plus twice the clearance "C."

Table of Proportions of Worm Threads to Run in Worm Wheels

C. P.	pi.	D. P.	T.H.	С	ರ	S. P.	W. D.	Thirt	W. W.	ä
Jrcular Pitch	Threads per Inch	Dia- metrical Pitch	above Pitch- Line	Working Depth of Tooth	Clearance	of Space below Pitch Line	Whole Depth of Tooth		Widtin of Thread Tool at End	Width of Thread at Top
C. P. Inches	i.=_C. P.	D.P.= $\frac{\pi}{\text{C.P.}}$	$H.=\frac{I}{D.P.}$	$D_* = 2 \times \frac{1}{D.P.}$	$C = \frac{T}{10}$	S.=H.+C.	W. D.= D.+C.	$T = \frac{C.P}{2}$	W.= .31×C.P.	B. = .335 ×C. P.
	421	1.5708	.6366	1.2732	.1000	.7366	1.3732	1,0000	.6200	8029
-	-Ne-	1.7952	.5570	1.1141	.0875	-6445	1.2016	.8750	.5425	.5862
	eren -	2.0944	•4775	.0540	.0750	.5525	1.0299	.7500	-4650	.5025
_	+10	2.5133	•3979	.7958	.0625	.4604	.8583	.6250	.3875	.4187
	Η,	3.1416	.3183	9989*	•0500	.3683	9989*	•5000	.3100	•3350
-	-t:	4.1888	.2387	•4775	.0375	.2762	.5150	•3750	+2325	.2512
		4.7124	.2122	+4244	.0333	-2455	-4577	•3333	•2066	,2233
-	۵,	6.2832	.1592	.3183	.0250	.1842	•3433	.2500	•1550	.1675
-	53	7.8540	.1273	.2546	.0200	•1473	.2746	,2000	.1240	.1340
	۳,	9.4248	1901.	*2123	9910	.1227	.22288	9991.	.1033	41117
_	32	10.9956	6060*	0181.	.0143	.1052	.1962	.1429	9880	.0057
-	4	12.5664	9620	1651.	.0125	.0921	.1716	.1250	.0775	.0838
_	43	14.1372	1070.	.1415	1110.	.oS18	.1526	11111	6890.	•0744
_	Ŋ	15.7080	•0637	.1273	.0100	.0737	.1373	•1000	.0620	0290
	9	18.8496	.0531	1901.	.0083	4190	•1144	,0833	7150	.0558
_	_	21.9911	•0455	0160	1700.	.0526	1860.	•0714	.0443	•0479
_	»	25.1327	\$650	9620.	2000.	•0400	.0858	•0625	.0388	0419
	6	28.2743	.0354	.0707	.0055	.0409	.0763	.0555	.0344	.0372
	01	31.4159	.0318	.0637	.0050	.0368	2890	.0500	.0310	.0335
_	12	37.6992	*0202	.0530	.0042	.0307	.0572	.0416	.0258	.0279
	14	43.0824	40227	•0454	.0036	.0263	.0490	.0357	.0221	.0239
_	91	50.2055	6610	.0398	.0031	.0230	.0429	.0312	£010°	.0200
_	28	56.5488	9210.	.0352	.0025	.0201	.0377	.0255	.0172	.0186

or Multiple unreads, divide the sizes given in the table for the same lead, by 2 for double, 3 for triple, 4 for quadruple threads, To find the Pitch Diameter (given pitch and number of teeth)

pitch diameter. — Note. — Diameter of Hob equals diameter of Worm plus twice the clearance "C." Rule. - Multiply the number of teeth by the circular pitch and divide by 3.1416. Example. — Number of teeth, 96; Circular pitch, 4.

SPEEDS AND FEEDS FOR GEAR CUTTING

(CINCINNATI GEAR CUTTING MACHINE Co.)

Range of different sizes of machines as follows:

No. 3. Up to and including 4 diametral pitch.

No. 4. Up to and including 3 diametral pitch.

No. 5. Up to and including 2 diametral pitch. No. 6. Up to and including $1\frac{3}{4}$ diametral pitch. No. 7. Up to and including 1 diametral pitch.

FOR CARBON STEEL CUTTERS RUNNING AT A PERIPHERAL SPEED OF 35 FEET PER MINUTE ON CAST IRON AND 30 FEET PER MINUTE ON STEEL

al Pitch ear	Feed in In. per Finishing in one				Feed in In. per Min. for Roughing Cut .o10 to .030 in. Left for Finishing Cut			Feed in In. per Min. for Finishing Cut				
Diametral P of Gear	Cast Iron	Soft Steel	Hígh Car- bon Steel	Nick- el Steel	Cast Iron	Soft Steel		Nick- el Steel	Cast Iron	Soft Steel	bon Steel	Nick- el Steel
1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	218 213 213 316 316 316 316 316 316 316 316 414 414	116 2 2 2 121212 2 2 1212 2 2 3 17 6 3 17 6	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	I I 4 4 4 4 5 5 5 5 5 5 5 5 5 5 5 5 5 5	1 5 5 1 6 3 3 8 4 1 6 4 1 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	I I I I I I I I I I I I I I I I I I I	I I I I I I I I I I I I I I I I I I I	21 6 2 1 6 1 8 2 1 6 1 8 2 1 6 1 8 2 1 6 1 8 2 1 6 1 8 2 1 8	1.58 1.68 3 1.8 1.8 1.8 1.8 1.8 1.8 1.8 1.8 1.8 1.8	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	1 1/4 1 1/4 1 1/5 8 2 1/6 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2

FOR HIGH SPEED STEEL CUTTERS RUNNING AT A PERIPHERAL SPEED OF 55 FEET PER MINUTE ON CAST IRON AND SO FEET PER MINUTE ON STEEL

	PER MINUTE ON CAST IRON AND SO FEEL PER MINUTE ON STEEL											
al Pitch	Feed in In. per Min. Finishing in one Cut			Feed in In. per Min. for Roughing Cut .o10 to .030 in. Left for Finishing Cut			Feed in In. per Min. for Finishing Cut					
Diametral Pitch of Gear	Cast Iron	Soft Steel	High Car- bon Steel	Nick- el Steel	Cast Iron	Soft Steel	High Car- bon Steel	Nick- el Steel	Cast Iron	Soft Steel	High Car- bon Steel	Nick- el Steel
I 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	21163 3387 3177 3177 3177 3176 4414 4776 5178	216 212 212 212 212 212 213 216 316 317 316 444	218 2 1 2 1 2 2 2 2 2 2 3 1 6 3 1 7 6 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4	1105000B 11111111 2 2 2 2 2 3 3 3 3 5 5 5 5 5 5 5 5 5 5 5	2 1 2 2 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3	1 5 2 2 1 3 8 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4	1 2 2 1 6 1 6 1 6 1 6 1 6 1 6 1 6 1 6 1	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	116 6 8 9 6 6 8 9 5 7 7 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8	2 3 4 56 6 6 6 6 6 6 6 6 8 8	1151 man 16 lb alterior and red order of the control of the contro	2 2 3 4 5 5 5 5 6 6 6 6 6 8 8 8

MILLING AND MILLING CUTTERS

MILLING MACHINE FEEDS AND SPEEDS

THE determining of the proper feeds of milling cutters in the past was usually a matter of guesswork, or experience, as a good many would term it, no absolute rule of any kind having ever been established.

A guide for determining the proper feed of milling cutters is found

in ascertaining the thickness of the chip per tooth of the cutter.

Taking, for example, an average size milling cutter working in cast iron, say $2\frac{1}{2}$ inches diameter, 3 inches long, with eighteen teeth, which is quite commonly used, and it will be found that the thickness of the chip per tooth is quite small, resulting in .oo18 inch, with a table feed of 2 inches per minute. This is entirely too slow. Now, comparing this cut of .oo18 inch with a lathe tool cut, it will be seen that such a chip in a milling cutter is much smaller and is far more injurious to the cutter than a heavier feed, since the cutting edge of a tool will hold up longer in cutting into the metal instead of scraping it.

A cutter is very seldom ruined by the feed, but is generally ruined by overspeeding it. For instance, with a cutter of thirty teeth with a table feed of .300 inch per revolution, the chip per tooth will then

only be $\frac{300}{30T}$ = .010 inch thick — still quite a light cut when comparing it with a lathe tool chip. Hence in many cases of milling, if the feeds are guided by the thickness of chip per tooth, a much faster feed would be used, since it is evident that the heaviest feeds, comaratively, give only a thin chip per tooth.

CUTTING SPEEDS

The Brown & Sharpe Mfg. Co. recommends a cutting speed of 65 feet per minute for carbon and 80 to 100 feet per minute for high speed milling cutters under average conditions. On soft cast iron, having a tensile strength of about 13,000 pounds — the feed recommended is 0.148 inches per revolution or about 9 inches per minute for carbon cutters. With a medium cast iron of about 23,000 pounds tensile strength, the same speed is maintained but the feed reduced about $\frac{1}{3}$ or to 6 inches per minute. For high speed cutters the feed can run up to 0.26 inch per revolution on the softer iron.

On steel of 65,000 tensile strength, with a cut 6 inches wide by $\frac{8}{16}$ inch deep, a feed of 16 inches per minute can be maintained for long periods. At 60 revolutions per minute and a feed of 0.262 per revolution 18 cubic inches per minute was removed with 21 horsepower.

It is not always advisable to maintain the highest cutter speeds as a slower speed and heavier feed will prevent vibration and chatter. These are not maximum results but can be attained under regular working conditions. The horsepower required for removing a cubic inch of metal per minute on the milling machine may be safely considered as 1\frac{1}{4} horsepower for steel and \frac{3}{4} horsepower for cast iron.

THE ACTION OF A MILLING CUTTER

Experiments carried on with cutters at the works of the Cincinnati Milling Machine Company and extending over several years, have led to results of general interest. These tests covered milling cutters

of various types.

The action of the ordinary milling cutter is not a true cutting action, as it is commonly understood. By a true cutting action is meant the driving of a wedge-shaped tool between the work and the chip and, although this definition is not based on a generally accepted meaning of the term, it is believed that it expresses fairly well what most mechanics understand by cutting. Practically all milling cutters have their teeth radial and this, of course, excludes the possibility of driving a wedge between chip and work. The tooth compresses the metal until it produces a strain great enough to cause a plane of cleavage at some angle with the direction of the cutter. It then begins to compress a new piece, push it off, and so on. This at least seems to be the action of the cutter, judging by the form of the chips. These chips are in the form of needles or small bars.

The chip taken by a milling cutter varies very materially from those taken by a lathe or planer tool. These latter tools make chips of uniform section, whereas the section of a milling chip increases

from zero to a maximum.

Fig. 1 shows a milling chip as it would appear, if no compression or distortion took place. The proportions are very much exaggerated, so as to bring its typical shape clearer into view. The width A B at the top is equal to the feed per tooth. The height B C is the depth of cut. The length B D is the width of cut. The section M N O P, shown halfway on the chip, is a normal section and a measure of the amount of work which was done at the time the cutter passed the point M.

Fig. 2 shows the action of a milling cutter, with center O, when the cutter is rotating and the work is feeding at the same time. The tooth AB sweeps through the path BC. When the point B has reached the position B_1 , a new tooth starts cutting. By this time O has advanced to position O_2 , and the new tooth A_2B_2 is not yet in a vertical position, when the point B_2 touches the work. When the

cutter revolves, this point B_2 must penetrate into the work and compress the metal of the work. The result will be spring in the arbor. When this spring has assumed certain proportions, the blade or tooth

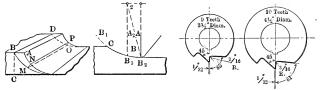


Fig. 1—Chip As- Fig. 2.—Action sumed to be produced by Cutter without distortion

Fig. 3. — Coarse pitch Milling Cutter

begins to remove a chip. This may be assumed to take place in the position B_3 , the tooth simply gliding over the work from B_2 to B_3 . This action must necessarily be very harmful to the cutter, and it was believed that this, perhaps more than any other action of the cutter, caused its dulling. It would be especially severe with light cuts, as a relatively small amount of spring would allow the point B_2 to travel through a large arc. It would be quite possible that a tooth should fail entirely to take a chip, and that the succeeding tooth would take a chip of double the amount.

This peculiar action of the milling cutter is inherent in its construction and cannot be avoided. This question then is how to

minimize these harmful results.

Another feature, which limits the ability of a milling cutter to remove metal, is the proportion between the chip to be removed and the amount of space between two adjoining teeth. Such a limitation does not exist with lathe or planer tools, where the chips have unlimited space in which to flow off.

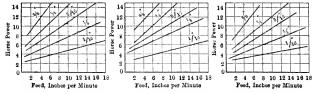
FORM OF TOOTH IN THE NEW CUTTER

The foregoing considerations led to a gradual evolution of spiral milling cutters. At first, the number of teeth of spiral mills was only slightly diminished, as it was thought that some element which was not considered might affect the result. Gradually the spacing was increased and the cutters, as now used, have taken the forms shown in Fig. 3.

Two standard sizes are used, although other sizes are required for special cutters and special gangs. The standard diameters are $3\frac{1}{2}$ inches and $4\frac{1}{2}$ inches. The $3\frac{1}{2}$ -inch diameter cutters are made with nine, and the $4\frac{1}{2}$ -inch diameter cutters with ten teeth, which corresponds to a spacing of about $1\frac{1}{4}$ inches. The point of the tooth has a land of $3\frac{1}{2}$ inch, and the back of the tooth forms an angle of 45 degrees with the radial line. The chip space is approximately four times as great as in the usual standard cutter of the present time and is formed with a $\frac{\pi}{4}$ -inch radius at the bottom.

RESULTS OF TESTS

Very satisfactory results were obtained with these cutters. Figs. 4, 5, and 6 show the results of tests made with cutters with $\frac{5}{8}$ -inch, $\frac{3}{4}$ -inch and $\frac{1}{8}$ -inch spacing. Cuts were taken on cast-iron test



Milling Cutters and their Efficiency

blocks. It will be noticed that the same amount of power is required to take a cut $\frac{1}{4}$ -inch deep and with 10.4 feed with a cutter of $\frac{5}{8}$ -inch pitch, and a cut $\frac{1}{4}$ inch deep and with 13.5 feed but with a cutter $\frac{1}{8}$ -inch pitch.

Therefore there is a large increase in the amount of metal which can be removed with the same amount of horsepower, by using these wide-spaced cutters. It was also found that for roughing on the ordinary work in the shop a cutter with the wider-spaced teeth would remain sharp for a longer period, notwithstanding that feeds had been increased.

THE FINISH OF THE WORK

It is a common belief that better finish can be obtained with teeth closely spaced, but experience with the wide-spaced cutter shows that there is no ground for this belief. The grade of finish may be expressed by the distance between successive marks on the work. marks are revolution marks and not tooth marks. It is practically impossible to avoid these revolution marks. They are caused by the cutter not being exactly round or quite concentric with the hole, by the hole not being of exactly the same size as the arbor, by the arbor not being round, by the straight part of the arbor not being concentric with the taper shank, by the taper shank not being round or of the same taper exactly as the taper hole in the spindle, by this taper hole being out of line with the spindle, by looseness between the spindle and its bearings, etc. Each of these items is very small in any good milling machine; yet the accumulation of these little errors is sufficient to cause a mark, and this mark needs to have a depth of only a fraction of a thousandth of an inch to be very plainly visible. As these marks are caused by conditions which return once for every revolution of the cutter, it is plain that the spacing of the teeth can have no effect on the distance between them and, therefore, on the grade of finish. This has been proven by actual tests.

THE CHIP BREAKER

It is generally believed that for finishing alone a milling cutter should be used without chip breakers, the effect of the chip breaker being to scratch the surface. To overcome this trouble, chip break-

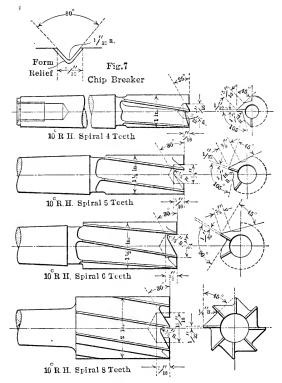


Fig. 8. - Taper Shank End Mills

ers are made as shown in Fig. 7 with clearance at both corners. This prevents the tearing up of metal with the result that a cutter with these chip breakers produces as good a finish as one without chip breakers.

END MILLS

Fig. 8 shows the end mills which are now considered standard by the Cincinnati Milling Machine Company and which fill practically all requirements. They are made in sizes of $\mathbf{1}$ inche, $\mathbf{1}$ inches, $\mathbf{1}$

inches and 2 inches in diameter, the smallest with four, and the largest with eight teeth. In order to preserve the strength of the teeth it is necessary to mill the back of the teeth of the three smaller sizes with two faces. Their action is remarkably free. A 2-inch taper shank end mill milled a slot $\mathbf{1}_{16}^{-1}$ inches deep in a solid block of cast iron at a rate of 6 inches per minute. The block was

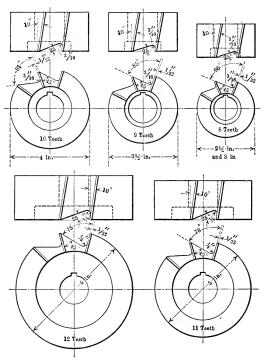


Fig. 9. - Spiral Shell Cutters

clamped to the table of the milling machine and the knee was fed upward. The same cutter would remove from the end of the casting a section $1\frac{1}{2}$ inches wide and $1\frac{1}{2}$ inches deep. Under the latter conditions, the chips would free themselves from the cutter and these chips were rolled up in pieces much like the chips obtained from a broad planer tool, when taking a finishing cut. This cut was taken with a feed of 11 inches per minute. Another similar cut, but 1 inch and $1\frac{1}{8}$ inches in section, was taken with a feed of 33 inches per minute.

Fig. 9 shows the shell end mills of the wide-spaced type, which are now considered standard for their use by the Cincinnati Milling Machine Company, and Figs. 10 and 11 show the side mills.

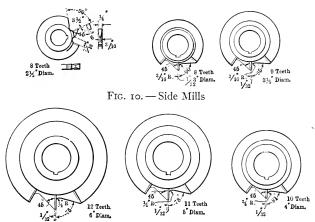


Fig. 11. — Side Mills

When milling steel, a heavy flow of oil on a milling cutter, forced by means of an oil pump, is just as essential as the great volume of oil which is used on automatic screw machine tools, which would not hold up one-half hour if not so flushed. The life of a milling cutter amply lubricated will be materially prolonged and it will be capable of standing a much heavier feed.



LEADS OF B. & S. CUTTER SPIRALS

The leads of the Brown & Sharpe Cutter Spirals are as follows:

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Diam. of Cutter	Lead	Diam. of Cutter	Lead
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MILLING MACHINE

To find the angle for cutters of a larger diameter than given in the diable, make a drawing as shown in the diagram; the angle b being a right angle. Let b c equal the circumference. Let a b equal the pitch. Connect a by a line, and measure the angle a with a protractor; or divide the circumference by the lead and the quotient will be the tangent of the angle. Find the angle in a table of tangents.	Diameter of Mill, Cutter, or Drill to be Cut Inches Values Given Under Diameters are Angles in Degrees	8 4 8 2 2 2 2 2 2 2 2 2 2 3 3 3 3 3 3 3 3 3	1.25 17½ 32‡ 28 38½ 28 38½ 28 38½ 28 38½ 28 38½ 28 38½ 28 38½ 28 38½ 28 38½ 28 34‡ 28 34‡ 43‡ 28 34‡ 43‡ 34 43‡ 34 34 34
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Table of Pitches and Approximate Angles for Cutting Spirals on the Universal

MILLING MACHINE

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MILLING HEART-SHAPED CAMS

ONE method of producing heart-shaped cams is as follows:

Lay out the curve of the cam roughly, as in Fig. 1. Drill and remove the outside stock, being sure to leave sufficient stock to overcome errors in laying out. Put the cam on the nut arbor and tighten securely. If the roll of the cam is \(\frac{1}{3}\) radius, select a milling cutter having the same radius, as the roll of the cam must come to the lowest point, which it would not be able to do if a cutter of a smaller radius than that of the roll were used. It would also make a difference to the other points on the curve of the cam, which is not quite so apparent at first glance.

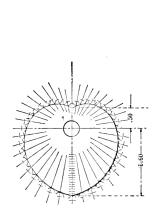


Fig. 1. — Method of Laying Out Cam

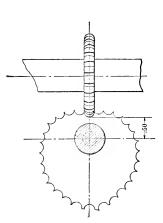


Fig. 2. — Position of Cam and Cutter when Commencing to Mill

Selecting an Index

THE next operation is to place the cam between centers on the milling machine, having the cutter in line with the vertical radius of the cam, at its lowest point. Next choose an index circle which will give a division of the cam such that the rise of each division will be in thousandths of an inch, if possible. For this cam take a circle which will give 200 divisions. As this will make 100 divisions on a side, the rise of each division will be 0.011 of an inch. Now raise the table to the required hight, starting at the lowest point of the cam, and mill across, as in Fig. 2.

Moving the Table

For the other cuts lower the table o.o.i each time, and revolve the cam one division until the highest point of the cam is reached, then raise the table o.o.i for each division of the cam.

When the cam comes from the milling machine there will be found to be small grooves left between the cuts. These may be easily removed by smoothing off with a file without impairing the accuracy of the cam.

Most screw-machine cams can be made in this manner, and they will be found to be more accurate than if laid out and filed to the line, and also much easier to make after one has become accustomed to the method.

MILLING CAMS BY GEARING UP THE DIVIDING HEAD

By the method here shown, cams of any rise may be milled with the gears regularly furnished with the milling-machine.

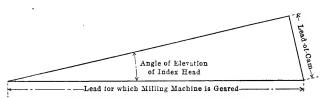


Fig. 3.— Diagram for Angle of Index Head

With the head set vertically the lead of the cam would be the same as the lead for which the machine is geared, while with the head horizontal and the milling spindle also, a concentric arc, or rest, would be milled on the cam, regardless of how the machine was geared. By inclining the head and milling spindle, we can produce any lead on the cam less than that for which the machine is geared.

The method of finding the inclination at which to set the index head is shown in Fig. 3, and is simply the solution of a plain right-angled triangle, in which the hypothenuse represents the lead of the machine, and one of the other sides represents the lead we wish to produce on the cam. By dividing the latter by the former we get the sine of the angle of inclination.

Take for illustration a plate cam having \frac{1}{8}-inch rise in 300 degrees.

$$\frac{360}{300} \times \frac{1}{8} = 0.15,$$

which is the lead we want on the cam, while the slowest lead for which the B & S. machine can be geared is 0.67

$$\frac{0.15}{0.67} = 0.224.$$

Consulting a table of sines, we find 0.224 approximates closely the sine of 13 degrees, which is the angle at which to set the head, and if the milling spindle is also set at the same angle, the edge of the cam will be parallel with the shaft on which it is to run. Fig. 4 shows a milling-machine set for this job.

When a cam has several lobes of different leads, we gear the machine up for a lead somewhat longer than the longest one called for in that cam, and then all the different lobes can be milled with the one setting of gears, by simply altering the inclination of head

and milling spindle for each different lead on the cam.

If the diameter of the cam and the inclination of the head will admit, it is better to mill on the under side of the cam, as that brings the mill and the table nearer together and thus increases rigidity, besides enabling us to easily see any lines that may be laid out on the flat face of the cam. Also the chips do not accumulate on the work.

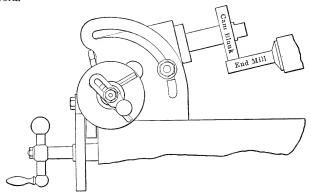


Fig. 4. - Dividing Head Set for Cam Milling

The work is fed against the cutter by turning the index crank, and on coming back for another cut we turn the handle of the milling-machine table. As a result the work will recede from the cutter before the cam blank commences to turn, owing to back lash in the gears, thus preventing the cutter from dragging over the work while running back.

In this way we use to advantage what is ordinarily considered a

defect in machine construction.

The milling-machine, when used as shown in Fig. 4, will be found to be more rigid than when the head is set in the vertical position, and the cams will work more smoothly on account of the shearing action of the cutter.

One possible objection to the method here advocated is the necessity of using, in some cases, an end mill of extra length of tooth. In practise, an end mill $\frac{7}{8}$ -inch diameter and with a $3\frac{1}{2}$ -inch length of tooth is not unusual; but the results in both speed and quality will be found entirely satisfactory.

TABLES OF SETTINGS FOR MILLING SCREW MACHINE CAMS

COMPUTED BY THE CINCINNATI MILLING MACHINE CO.

On the preceding pages an explanation is given of the methods of computing the angle at which to set the dividing head and milling head for cutting spiral screw machine cams or other cams of similar form to any desired lead. For leads below 0.6 inch the method referred to will be of direct service, but where the lead is greater than 0.6 inch the following tables can be used to great advantage as these give at once the settings of dividing head and vertical milling attachment for leads from 0.6 inch to 6 inches.

These tables give all the information necessary and it only remains for the milling machine operator to select the lead of the desired cam from the tables and set up for the corresponding change gears and

angles.

In setting the vertical milling attachment read the angle direct from the dial. Example: if the angle given in the table is $39\frac{1}{2}$ degrees, set the spindle $39\frac{1}{2}$ degrees from its vertical position.

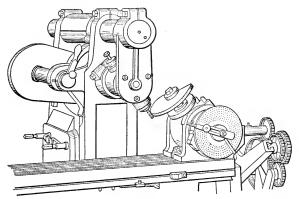


Fig. 5. - Milling Cams

In setting the dividing head, subtract the angle in the table from 90 degrees. The difference represents the angle to which the dividing head spindle must be raised from the horizontal position.

Example: The angle given in the table is 30\frac{1}{2} degrees. 90 degrees -

39½ degrees equals 50½ degrees.

Set the dividing head spindle 50½ degrees up from the horizontal position. This angle is read direct from the dial. The set up is shown in Fig. 5.

The tables may of course be used in connection with the cutting of

any other similar cams.

Lead	Gear on Worm	ıst Intermediate	2nd Intermediate	Gear on Screw	Angle	Lead	Gear on Worm	1st Intermediate	2nd Intermediate	Gear on Screw	Angle	Lead	Gear on Worm	1st Intermediate	2nd Intermediate	Gear on Screw	Angle
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-751	Lead	Gear on Worm	1st Intermediate	2nd Intermediate		Angle	Lend	u c	ıst Intermediate	2nd Intermediate	Gear on Screw	Angle	Lead	Gear on Worm	rst Intermediate	and Intermediate	ä	Angle
.,99 -4 ,2 32 35 41 2 .049 24 00 32 35 10 .099 24 ,2 20 100 132	.751 .752 .753 .755 .755 .755 .755 .755 .755 .761 .762 .763 .764 .765 .765 .766 .777 .778 .777 .778 .777 .778 .777 .778 .777 .778 .779 .778 .778	244 244 244 244 244 244 244 244 244 244	726 726 726 726 726 726 726 726 726 726	28 1 2 2 4 2 3 2 4 5 4 5 4 5 4 5 4 5 4 5 4 5 4 5 5 6 5 6	Too Too	16 20 21 31 32 31 31 31 31 31 31 31 31 31 31 31 31 31	\$01 \$02 \$03 \$045 \$06 \$06 \$06 \$06 \$06 \$06 \$06 \$06 \$06 \$06	244 244 244 244 244 244 244 244 244 244	724666726672266872276872266672266672266678722664	144221248442864442846828423822332233423422223332222233222223223222322232223223	86 86 86 86 86 86 86 86 86 86 86 86 86 8	$\begin{array}{cccccccccccccccccccccccccccccccccccc$.851 .851 .853 .854 .855 .856 .857 .860 .861 .863 .863 .864 .863 .866 .871 .872 .873 .874 .875 .877 .878 .877 .878 .877 .878 .887 .888 .888 .888 .888 .888 .889 .890 .890	21, 24, 24, 24, 24, 24, 24, 24, 24, 24, 24	6427286466672466466466666666666666666666	2.13 2.32 3.24 4.24 2.24 2.24 2.24 2.24 2.2	100 86 100 86 100 86 100 100 100 100 100 100 100 100 100 10	19 43 756 5725 756 46 4515 3 424 1515 2 1515 152 1515 15 15 15 15 15 15 15 15 15 15 15 1

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Lead	Gear on Worm	1st Intermediate	2nd Intermediate	Gear on Screw	Angle	Lead	Gear on Worm	ıst Intermediate	2nd Intermediate	Gear on Screw	Angle	Lead	Gear on Worm	1st Intermediate	2nd Intermediate	Gear on Screw	Angle
.900 .901 .902 .903 .904 .914 .915 .916 .917 .918 .919 .919 .919 .921 .923 .924 .926 .927 .928 .929 .928 .929 .930 .931 .931 .934 .931 .934 .939 .931 .934 .939 .939 .939 .939 .939 .939 .939	24 24 24 24 24 24 24 24 24 24 24 24 24 2	5672772277227722772277227722772277227722	24	100 86 100 86 100 86 100 1	295 144 33 32 1 1 1 1 1 1 1 2 1 2 2 4 2 2 3 2 4 2 2 3 2 4 2 2 3 2 4 2 2 2 2	950 -951 -952 -953 -954 -955 -950 -950 -960 -963 -964 -965 -966 -966 -967 -973 -974 -975 -975 -978 -980 -980 -980 -980 -980 -980 -980 -98	2 1 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4	75666666666666666666666666666666666666	32448822448842232442842322442844232244232442322442322222222	1000 1000	$\begin{array}{c} 2 & 2 & 4 & 12 & 12 & 12 & 12 & 12 & 12$	1.000 1.001 1.002 1.003 1.005 1.006 1.007 1.006 1.007 1.011 1.012 1.014 1.015 1.014 1.015 1.014 1.015 1.014 1.015 1.014 1.015 1.014 1.015 1.014 1.016 1.017 1.018 1.019 1.019 1.019 1.021 1.021 1.022 1.023 1.024 1.025 1.026 1.027 1.028 1.029 1.030 1.031 1.032 1.033 1.034 1.034 1.035 1.036 1.037 1.038 1.039 1.039 1.039 1.039 1.039 1.039 1.039 1.039 1.039 1.039 1.039 1.039 1.049 1.	24 248 248 244 244 244 244 244 248 244 244	866 566 566 566 566 566 566 566 566 566	242 244 244 244 244 244 244 244 244 244	1000 1000	13 ¹ / ₂ 16 13 15 ¹ / ₂ 12 ¹ / ₂ 12 16 11 ¹ / ₂ 14 ¹ / ₂ 11

Lead	Gear on Worm	rst <u>.</u> Intermediate	2nd Intermediate	Gear on Screw	Angle	Lead	Gear on Worm	ıst Intermediate	2nd Intermediate	Gear on Screw	Angle	Lead	Gear on Worm	ıst Intermediate	2nd Intermediate	Gear on Screw	Angle
										0.0							
1.050 1.052 1.054 1.058 1.058 1.068 1.066 1.066 1.074 1.074 1.074 1.076 1.082 1.082 1.082 1.088 1.088 1.088 1.088 1.088 1.088 1.088 1.090 1.092 1.104	24 24 24 24 24 28 24 24 24 24 24 24 24 24 24 24 24 24 24	786 766 66 766 676 74 746 666 666 7666 666	28 4 4 4 8 8 4 4 4 4 4 4 4 4 4 4 4 4 4 4	100 86 100 86 100 100 100 100 86 100 100 100 100 100 100 86 86 100 100 100 86 86 100 100 100 86 86 100 100 86 86 100 86 86 86 100 86 86 86 86 86 86 86 86 86 86 86 86 86	$\begin{array}{c} 35^{\frac{1}{2}} \\ 12 \\ 17 \\ 27 \\ 16^{\frac{1}{2}} \\ 26^{\frac{1}{2}} \\ 39^{\frac{1}{2}} \\ 15^{\frac{1}{2}} \\ 41 \\ 25^{\frac{1}{2}} \\ 28^{\frac{1}{2}} \\ 28^{\frac{1}{2}} \end{array}$	1.150 1.152 1.154 1.156 1.158 1.160 1.162 1.166 1.170 1.172 1.174 1.178 1.180 1.182 1.184 1.188 1.188 1.188 1.188 1.189 1.190 1.192 1.194 1.196 1.200 1.201 1.201 1.201 1.201 1.212 1.214 1.216 1.212 1.214 1.220 1.222 1.224 1.223 1.224 1.225	218 24 1 1 2 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	566 644666 644266 666 64446 2244422246 626 26446 675 566 676 576 676 677 777 777 867 86426 776 677 777 867 86426 776 677 777 867 86426 776 777 777 867 867 867 867 867 867 86	2 4 4 8 2 2 4 4 4 2 2 2 6 4 4 4 4 4 4 4 4 4 4 4	86 86 86 100 100 100 100 100 100 100 100 100	$\begin{array}{c} 16 \\ 16 \\ 16 \\ 16 \\ 16 \\ 16 \\ 16 \\ 16 $	1.250 1.252 1.254 1.258 1.260 1.262 1.262 1.264 1.263 1.272 1.274 1.274 1.278 1.278 1.284 1.284 1.284 1.289 1.292 1.294 1.300 1.300 1.300 1.300 1.301 1.312	248 24 1 2 1 2 2 8 2 2 2 2 8 8 8 8 2 2 4 0 2 4 4 2 2 4 4 4 4 4 4 4 4 4 4 4	646 646 656 657 656 656 656 656 656 656 656 65	48	100 100 100 100 100 100 100 100 100 86 100 100 100 100 100 100 100 100 100 10	$\begin{array}{cccccccccccccccccccccccccccccccccccc$
1.134	24 24	64	28	86	2 I 1	1.236	24	72	40	100	22	1.336	32	56	24	100	13
1.138	24	64	32 28		18½ 21	1.238	28 24	86 72	40 40		18 21½	1.338	32 24	56 72	24 44		12½ 24
I.140 I.142	24	64 64	32		35	1.240	28	86	40		$17\frac{1}{2}$	1.340	28	64	32	100	161
1.144	24	56 86	24	86	17	1.244	24	72	40	100	2 I	1.344	24	64	32	86	152
1.146	24	86 64	44 32		2I 17	I.246 I.248	32 28	72 86	40 40	100	45 [‡]	1.346	32 32	56 56	24 24		II 101/2
1.140	24	04	32	100	1/	1.240	20	00	40	100	102	1.340	32	30	-4	100	102
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Lead	Gear on Worm	ıst Intermediate	2nd Intermediate	Gear on Screw	Angle	Lead	Gear on Worm	rst Intermediate	2nd Intermediate	Gear on Screw	Angle	Lead	Gear no Worm	ıst Intermediate	2nd Intermediate	Gear on Screw	Angle.
1.352 1.354 1.356 1.356 1.356 1.356 1.360 1.362 1.366 1.366 1.366 1.370 1.372 1.374 1.376 1.378 1.388 1.390 1.388 1.390 1.388 1.390 1.391 1.398 1.402 1.411 1.416 1.408 1.408 1.402 1.411 1.416 1.418 1.422 1.424 1.424 1.426 1.438 1.438 1.438 1.438 1.438 1.438 1.438 1.438 1.438 1.438 1.438 1.438 1.438 1.438 1.438 1.438 1.438 1.438 1.449 1.441 1.441	2488 4 4 4 4 8 8 7 4 4 5 8 8 4 8 6 8 8 4 8 8 8 8 8 8 8 8 8 8 8 8	564 664 664 726 664 786 664 786 664 786 664 786 664 786 664 786 664 786 7686 76	41 32 41 28 32 32 32 32 32 44 41 32 32 48 40 41 42 44 44 44 44 44 44 44 44 44	86 86 72 100 86 100 86 100 100 86 100 100 86 100 100 86 100 100 100 100 100 100 100 100 100 10	$\begin{array}{c} -10 \\ 15 \\ 15 \\ 12 \\ 12 \\ 12 \\ 12 \\ 12 \\ 12$	1.450 1.452 1.454 1.456 1.456 1.456 1.462 1.466 1.468 1.472 1.474 1.476 1.476 1.476 1.476 1.476 1.476 1.488 1.488 1.488 1.488 1.488 1.489 1.492 1.496 1.492 1.500 1.	340342400400000404004040404040540540540545444410544440056014604054040	8646642222222222222222426687222662222662220242424	4248000244001440000000000000000000000000	Too Too	13 1412 12 12 13 14 12 12 13 14 12 12 13 14 14 13 12 13 14 14 13 12 13 14 14 13 14 14 13 14 14 14 15 15 18 18 18 18 18 18 18 18 18 18 18 18 18	1.550 1.552 1.554 1.556 1.558 1.562 1.568 1.568 1.572 1.574 1.576 1.575 1.576 1.575 1.582 1.584 1.586 1.586 1.586 1.586 1.586 1.588 1.586 1.606 1.602 1.604 1.606 1.612 1.618 1.622 1.624 1.638	411111111111111111111111111111111111111	67264422662246676464664666666666666676667	248 0 2 4 4 4 8 0 2 4 4 8 4 2 4 4 4 2 4 4 4 2 4 4 4 2 4 4 4 4	100 86 100 100 100 100 100 100 100 100 100 10	204472149 237 12 215007 4623 4 2 15 15 15 15 15 15 15 15 15 15 15 15 15

Lead	Gear on Worm	ıst Intermediate	2nd Intermediate	Gear on Screw	Angle	Lead	Gear on Worm	ıst İntermediate	2nd Intermediate	Gear on Screw	Angle	Lead	Gear on Worm	ıst Intermediate	2nd Intermediate	Gear on Screw	Angle
1.650 1.652 1.654 1.656 1.658 1.666 1.668 1.672 1.676 1.678 1.682 1.682 1.686 1.682 1.686 1.682 1.690 1.692 1.693 1.704 1.704 1.704 1.712 1.712 1.712 1.712 1.712 1.712 1.712 1.724 1.725 1.736 1.738 1.736 1.738 1.738 1.736 1.738 1.736 1.738 1.736 1.738 1.736 1.738 1.736 1.738 1.736 1.738 1.746 1.744 1.746 1.748	280 244 242 288 228 228 248 248 288 248 228 248 24	646 72 72 72 6 72 72 6 72 72 6 72 72 6 72 72 6 72 72 6 72 72 72 6 72 72 72 72 72 72 72 72 72 72 72 72 72	41444444444444444444444444444444444444	86 36 100 100 86 86 86 86 100	1154 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4	1.750 1.752 1.752 1.754 1.755 1.756 1.758 1.766 1.768 1.776 1.776 1.778 1.778 1.778 1.786 1.788 1.789 1.799 1.799 1.799 1.799 1.799 1.808 1.808 1.812 1.814 1.818 1.818 1.822 1.824 1.828 1.833 1.834 1.838 1.838 1.838 1.838 1.838 1.838 1.838 1.838 1.838 1.838 1.848	325 8 2 2 2 8 8 4 4 5 8 4 4 5 8 4 5 8 5 4 5 8 5 4 5 8 5 7 4 5 8 5 7 4 5 8 5 7 4 5 8 5 7 5 8 6 7 5 7 6 8 7 6 7 6 7 6 7 6 7 6 7 6 7 6 7 6 7	86 00 72 86 72 72 86 72 72 86 72 72 86 72 72 86 72 72 86 72 72 86 72 72 86 72 72 86 72 72 72 72 72 72 72 72 72 72 72 72 72	45688 458 488 488 0 446 0 0 68866 42 4 44 2 488 48 4 4 4 4 4 4 8 0 8 8 4 4 4 4 2 4 4 6 2 5 4 4 6 5 5 5 2 3 4 4 4 3 4 4 4 4 4 4 4 4 4 4 4 4 4 4	100 866 866 866 866 100 72 866 100 866 866 866 866 866 866 866 866 866 8	116 20 6 6 12 13 18 12 13 18 12 13 18 12 13 18 12 13 18 18 18 18 18 18 18 18 18 18 18 18 18	1.850 1.852 1.854 1.856 1.858 1.860 1.858 1.860 1.870 1.870 1.874 1.876 1.874 1.876 1.878 1.884 1.890 1.902 1.902 1.904 1.903 1.904 1.904 1.904 1.903 1.904 1.904 1.903 1.904 1.903 1.904 1.904 1.903 1.904	0.00 1+1+1+0 1+0.00 1+0	64 56 64 64 64 64 64 64 64 64 64 6	444 444 444 444 444 444 444 444 444 44	100 566 100 72 86 100 86 100 86 100 86 100 86 100 100 100 100 100 100 100 10	6 1210 7 143 1 43 1 43 1 43 1 1 1 1 1 1 1 1 1 1

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1.952	Lead	Gear on Worm	ıst Intermediate	2nd Intermediate	Gear on Screw	Angle	Lead	Gear on Worm	1st Intermediate	2nd Intermediate	Gear on Screw	Angle	Lead	Gear on Worm	ıst Intermediate	2nd Intermediate	Gear on Screw	Angle
2.014 32 72 40 86 13 2.112 40 44 24 100 143 2.212 33 04 40 80 18 18 2.016 32 72 40 86 13 2.114 28 56 32 72 18 2.216 32 72 40 86 12 2.116 28 64 44 86 19 2.216 32 72 40 86 12 2.118 28 100 56 72 13 2 2.206 32 72 48 86 21 2 2.118 28 100 56 72 13 2 2.202 32 72 44 86 12 2.022 28 62 48 86 21 2 2.022 28 62 48 86 21 2 2.022 28 62 48 86 12 2 2.022 28 62 48 86 12 2 2.022 28 62 48 86 12 2 2.024 28 64 48 100 15 2.122 28 100 56 72 12 2 2.224 32 72 44 86 12 2.026 28 64 48 100 15 2.122 28 100 56 72 12 2 2.224 32 72 44 86 12 2.028 28 64 48 100 15 2.128 28 100 81 2 2.224 32 72 44 86 12 2.028 28 64 48 100 15 2.128 28 100 81 2 2.224 32 72 44 86 12 2.028 28 64 48 100 15 2.128 28 100 21 2 2.026 28 20 40 86 10 2.126 28 100 56 72 12 2.226 32 72 44 86 11 2 2.036 32 72 40 86 10 2.123 28 100 56 72 12 2.230 32 64 40 86 10 2.236 20 32 72 40 86 10 2 2.136 28 100 56 72 12 2.236 32 72 44 86 11 2 2.036 32 72 40 86 10 2 2.136 28 100 56 72 12 2.236 32 72 44 86 11 2 2.036 32 72 48 86 10 2.238 28 100 20 2 2 2.036 28 64 48 100 14 2.136 28 100 56 72 11 2 2.234 44 48 28 100 20 2 2.036 32 72 48 86 10 42 2.386 28 64 48 100 12 2.136 28 56 32 72 16 2.236 32 72 44 86 10 2.238 28 64 48 100 12 2.136 28 56 32 72 16 2.236 32 72 44 86 10 2.238 28 64 48 100 12 2.136 28 56 32 72 16 2.236 32 72 44 86 10 2.238 28 64 48 100 12 2.136 28 56 32 72 16 2.236 32 72 44 86 10 2.238 28 64 48 100 12 2.136 28 56 32 72 16 2.236 32 72 44 86 10 2.238 28 64 48 100 13 2.142 28 100 56 72 10 2.246 32 56 40 100 112 2.042 28 64 44 40 72 12 2.042 28 64 44 72 12 2.044 40 44 24 100 20 2 2.144 28 100 56 72 10 2.246 32 2.44 32 56 40 100 112 2.046 28 64 48 100 13 2.142 28 100 56 72 10 2.246 32 2.44 32 56 40 100 112 2.046 28 64 48 100 13 2.142 28 100 56 72 10 2.246 32 2.44 32 56 40 100 112 2.046 28 64 48 100 13 2.142 28 100 56 72 10 2.246 32 64 40 100 112 2.046 28 64 40 86 64 40 72 11 2 2.046 28 64 40 86 64 40 72 11 2 2.046 28 64 40 86 64 40 72 11 2 2.046 28 64 40 86 64 40 72 11 2 2.046 28 64 40 86 64 40 72 11 2 2.046 28 64 40 86 64 40 72 11 2 2.046 28 64 40 86 100 13 2.142 28 100 66 12 2.04	1.952 1.954 1.956 1.958 1.950 1.962 1.962 1.962 1.962 1.972 1.972 1.984 1.984 1.986 1.982 1.982 2.092 2.092 2.092 2.092 2.092 2.094 2.092 2.093 2.094	40 24 3 4 3 4 3 4 3 4 3 4 3 4 3 4 3 4 3 4	64266466444456456444566564465866786786777656444244764764	448444444444444444444444444444444444444	100 100 100 100 100 100 100 100 86 86 86 86 86 86 86 86	$\begin{array}{c} 171^{\frac{1}{2}} \\ 217^{\frac{1}{2}} \\ 170$	2.052 2.052 2.056 2.058 2.050 2.062 2.062 2.064 2.072 2.074 2.076 2.072 2.074 2.086 2.088 2.080 2.082 2.082 2.083 2.080 2.082 2.084 2.086 2.088 2.100 2.090 2.092 2.094 2.104 2.114 2.114 2.115 2.114 2.115 2.114 2.120 2.122 2.124 2.128 2.130 2.130 2.132 2.134 2.138 2.138	288004442822222828282828282828888888888	64 646 647 726 647 727 727 727 644 445 564 100 640 100 672 100 640 100 672 672 100 670 670 670 670 670 670 670 670 670 6	44488 08 08 08 88 48 08 68 88 4 4 4 5 6 8 8 6 8 4 4 1 2 2 4 6 8 6 8 6 8 6 4 6 1 8 8 6 8 6 8 6 8 6 8 6 8 6 8 6 8 6 8	866 1000 1000 1000 1000 1000 1000 1000 1	$\begin{array}{c} 23\frac{1}{2} \\ 2123\frac{1}{2} \\ 215000000000000000000000000000000000000$	2.152 2.154 2.156 2.158 2.160 2.162 2.164 2.170 2.172 2.174 2.178 2.188 2.180 2.182 2.184 2.188 2.189 2.189 2.199 2.192 2.194 2.198 2.200 2.202 2.204 2.202 2.214 2.214 2.220 2.222 2.224 2.228 2.238 2.238 2.222 2.223 2.223 2.223 2.223 2.223 2.223 2.223 2.223 2.234 2.238	32 4 2 2 0 2 2 2 2 2 8 2 6 0 0 8 2 4 2 8 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	644446646464262626266664666624260442624262	44 44 44 44 44 44 44 44 44 44 44 44 44	1000 722 1000 866 1000 1000 1000 1000 1000 1000	120121141213

Lead	Gear on Worm	1st Intermediate	2nd Intermediate	Gear on Screw	Angle	Lead	Gear on Worm	ıst Intermediate	2nd Intermediate	Gear on Screw	Angle	Lead	Gear on Worm	ıst Intermediate	2nd Intermediate	Gear on Screw	Angle
2.250 2.255 2.260 2.255 2.270 2.275 2.270 2.285 2.285 2.305 2.315 2.320 2.315 2.325 2.330 2.345 2.340 2.355 2.355 2.355 2.365 2.365 2.365 2.365 2.365 2.365 2.375 2.385 2.390 2.2400 2.410 2.410 2.425	212 3188 8 2 8 4 4 4 2 4 4 8 8 0 8 4 1 8 4 1 8 1 2 8 8 0 6 8 8 4 2 8 1 8 8 1 8 8 1 8 1 8 1 8 1 8 1 8 1 8	72	448 3 3 3 2 2 3 4 4 4 8 0 0 0 0 2 2 6 8 8 8 0 1 5 8 8 8 8 8 6 6 6 6 8 8 6 6 6 6 8 8 4 8 4	86 86	10 14 ¹² 13 ¹² 13 12 ¹² 11 10 ¹² 12 ¹² 11 13 ¹² 12 ¹² 15 ¹² 18 ¹² 27 15 11 10 ¹² 18 ¹² 11 11 12 ¹² 18 ¹² 11 10 ¹² 18 ¹²	2.500 2.515 2.515 2.520 2.525 2.525 2.535 2.535 2.535 2.535 2.565 2.575 2.580 2.590 2.695 2.705 2.705 2.715 2.715 2.725 2.735	- 8 1 4 8 4 4 2 2 2 2 8 8 2 2 8 8 4 4 0 8 2 4 4 2 2 2 2 4 4 8 8 8 4 0 8 8 6 0 0 8 6 0 0 8 4 4 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8	48 100 40 44	4 4 4 4 4 2 3 2 4 4 4 4 4 4 8 2 2 4 4 5 4 4 4 4 4 8 2 2 4 5 4 4 4 4 4 8 2 2 4 5 5 4 4 4 2 3 4 4 5 5 4 6 2 2 4 5 5 4 4 5 5 4 4 5 5 4 6 6 2 3 4 5 5 4 4 5 5 5 5 5 5 5 5 5 5 5 5 5 5	722 72368868686868686868686868686868686868686	43 ¹ / ₂ 24 41 ¹ / ₂ 12 41 44 12 11 ¹ / ₂ 15 41 42 43 39 12 43	2.750 2.755 2.760 2.775 2.7760 2.7770 2.7785 2.7850 2.7850 2.8805 2.815 2.8205 2.815 2.8205 2.8400 2.855 2.8400 2.855 2.8400 2.855 2.8900 2.875 2.8900 2.8955 2.9000 2.9015 2.915 2.925 2.930 2.935 2.9000 2.945 2.955 2.9600 2.965 2.975 2.9800 2.965 2.975 2.9800 2.965 2.975 2.9800 2.985	28 448 40 0 40 448 428 40 228 80 24 448 40 40 40 40 40 40 40 40 40 40 40 40 40	644 4048 472 724 448 456 556 664 455 664 464 644 644 644 644	48 2 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4	72 100 86 56 72 72 72 72 72 72 72 72 72 72 72 72 72	$\begin{array}{c} 37^{\frac{1}{2}}\\ 38\\ 34\\ 111^{\frac{1}{2}}\\ 120^{\frac{1}{2}\frac{1}{2}}\\ 35^{\frac{1}{2}}\\ 35\\ 16\\ 18^{\frac{1}{2}}\\ 35^{\frac{1}{2}}\\ 127^{\frac{1}{2}}\\ 27^{\frac{1}{2}}\\ 35^{\frac{1}{2}}\\ 35^{\frac{1}{2}$
2.495	24	44	40	1	101	2.745	40	72	44		15	2.995	"			"	37

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Lead	Gear on Worm	ıst Intermediate	2nd Intermediate	Gear on Screw	Angle	Lead	Gear on Worm	ıst Intermediate	2nd Intermediate	Gear on Screw	Angle	Lead	Gear on Worm	ıst Intermediate	2nd Intermediate	Gear on Screw	Angle
3.000 3.005 3.010 3.015 3.020 3.025 3.030 3.035 3.045 3.055 3.070 3.055 3.070 3.105 3.105 3.105 3.115 3.125 3.130 3.145 3.140 3.145 3.155 3.160 3.165 3.175 3.185 3.185 3.195	408 4 4 4 3 2 4 4 6 5 8 8 4 4 4 0 8 8 4 4 4 4 8 2 3 4 6 6 4 8 8 4 4 4 4 8 8 8 4 4 4 4 8 8 8 4 4 4 4 8 8 8 4 4 4 4 8 8 8 4 4 4 4 8 8 8 4 4 4 4 8 8 8 4 4 4 4 8 8 8 4 4 4 4 8 8 8 4 4 4 4 8 8 8 8 4 4 4 4 8 8 8 4 4 4 8 8 8 8 4 4 4 8	56 72 100 40 44 64 44 44 40 72 44	568 448 408 444 440 484 444 440 484 444 44	72 86 86 86 56 86 72 72 72 86 86 86 86 72 86 72 86 72 86 72 86 72 86 86 72 86 86 72 86 86 86 72 86 86 86 86 86 86 86 86 86 86 86 86 86	42 1 1 2 1 2 2 3 3 3 3 3 4 1 2 1 2 2 3 3 3 3 4 1 2 7 2 3 1 6 2 1 2 3 1 2 1 2 2	3.250 3.265 3.265 3.270 3.275 3.285 3.285 3.305 3.305 3.315 3.315 3.345	444 322 480 414 482 324 402 403 424 403 424 403 404 404 404 405 405 406 407 407 408 408 408 408 408 408 408 408 408 408	48 100 44 56 56	480 410 410 410 410 410 410 410 41	64	28 13 2 1 2 1 2 1 1 1 1 2 2 3 1 1 7 2 2 4 2 2 1 1 7 2 2 4 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	3.500 3.515 3.510 3.515 3.510 3.515 3.520 3.535 3.530 3.535 3.545 3.550 3.570 3.580 3.580 3.580 3.580 3.580 3.615 3.620 3.615 3.620 3.615 3.620 3.635 3.640 3.645 3.650 3.650 3.650 3.650 3.675 3.600 3.615 3.675 3.710 3.715 3.720 3.715 3.720 3.715 3.720 3.735 3.710 3.745	288 400 284 440 440 448 440 440 448 440 440 448 440 440	48 472 404 48 456 566 41 48 400 648 400 648 400 656 448 400 656 448 400 656 48 400 656 48 400 656 48 400 656 4	444 456 448 444 4568 498 498 496 46566688 46548 486 486 486 486 486 486 486 486 486 4	72 72 72 86 86 86 72 72 72 86 72 72 100 86 72 72 100 100 100 56 86 72 72 100 100 100 100 100 72 100 100 100 100 100 100 100 100 100 10	12 $12\frac{1}{2}$ 26 $29\frac{1}{2}$ 12 $25\frac{1}{2}$

																	=
Lead	Gear on Worm	rst Intermediate	2nd Intermediate	Gear on Screw	Angle	Lead	Gear on Worm	ıst Intermediate	and Intermediate	Gear on Screw	Angle	Lead	Gear on Worm	ıst Intermediate	2nd Intermediate	Gear on Screw	Angle
3.7750 3.	340 4 4 38 5 0 4 5 6 6 5 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6	414 64 1 4 1 0 6 6 8 8 0 4 1 2 0 4 4 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	8 1 1 3 0 1 0 5 3 1 1 1 8 3 0 0 1 2 8 8 0 6 6 3 1 1 2 8 8 8 8 8 1 1 1 1 0 6 8 1 1 3 8 8 8 1 8 1 1 1 1 1 1 1 1 1 1 1	866 72 1000 866 72 1000 866 72 72 1000 866 72 72 1000 866 72 866 866 866 866 866 866 866 866 866 86	20111412 210111412 210111412 210111412 210111412 210111412 210112 200112	4.000 4.010 4.010 4.020 4.030 4.035 4.040 4.050 4.070 4.	28 0 6 6 0 4 0 2 4 4 4 5 6 0 0 0 4 4 4 4 5 5 4 5 6 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	1010 610 618 72 40 618 72 40 618 72 40 614 40 618 40 72 40 614 40 614 410 618 40 72 40 614 40 614 410 618 40 614 410 618 40 614 410 618 618 618 618 618 618 618 618 618 618	188 283 4886 1 1 1 4 4 8 8 18 8 18 8 18 6 6 4 1 4 18 6 6 4 8 8 6 8 4 8 16 6 8 18 16 8 18 16 8 18 16 8 18 16 8 18 16 8 18 16 8 18 16 8 18 16 8 18 16 8 18 16 8 18 16 8 18 16 8 18 16 8 18 16 8 18 16 8 18 18 18 18 18 18 18 18 18 18 18 18 1	727 727 727 7286 7286 866 868 868 868 868 868 868 722 644 867 727 756 868 727 756 868 727 756 756 756 756 756 756 756 756 756 75	316 - 10 - 10 - 10 - 10 - 10 - 10 - 10 -	1.250 1.260 1.205 1.205 1.205 1.205 1.205 1.205 1.205 1.205 1.205 1.310 1.305 1.310 1.315 1.320 1.315 1.320 1.325 1.320 1.325 1.320 1.325 1.320 1.325	256 4 5 4 4 2 2 2 4 4 4 2 2 5 7 6 2 2 3 5 5 5 3 4 4 5 4 4 5 5 2 4 6 2 2 4 4 5 3 2 4 4 2 2 5 7 6 2 2 2 3 5 5 5 3 4 4 5 5 4 4 5 5 4 6 2 2 4 4 5 3 2 5 6 2 2 2 2 5 5 5 5 5 5 5 5 5 5 5 5 5	4507 114 140688 16 146 1 16 1 16 1 16 1 16 1 16 1 16	68 18 668 14 6 1 1 18 6 6 8 1 18 14 16 16 18 18 17 18 18 18 18 18 18 18 18 18 18 18 18 18	86 86 86 86 86 86 72 72 86 72 72 86 100 86 72 72 72 72 72 72 72 72 72 72 72 72 72	2222475231919148083899613 1248122622722414132222141828502
	1									-							==

Lead	Gear on Worm	1st Intermediate	2nd Intermediate	Gear on Screw	Angle	Lead	Gear on Worm	1st Intermediate	2nd Intermediate }	Gear on Screw	Angle	Lead	Gear on Worm	1st Intermediate	2nd Intermediate	Gear on Screw	Angle
4.500 4.505 4.515 4.520 4.535 4.530 4.535 4.545 4.545 4.545 4.545 4.545 4.565 4.575 4.585 4.500 4.600 4.615 4.625 4.635 4.600 4.625 4.635 4.735	556 4486 456 464 23 688 8 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	444446466582474466746464646464646464646464646464646	3286444566886865445664444466644444666444446664444666444664446664468468	86 72 56 56 64 72 72 64 72 100 86	45 6 12 6 12 12 12 12 12 12 12 12 12 12 12 12 12	1.750 1.753 4.765 4.765 4.7765 4.7765 4.780 4.780 4.790 4.805 4.805 4.805 4.825 4.835 4.840 4.855 4.840 4.855 4.840 4.855 4.840 4.855 4.840 4.855 4.840 4.855 4.840 4.855 4.840 4.855 4.840 4.855 4.805 4.90	32 4 4 4 4 4 8 6 4 4 4 8 6 4 4 4 8 6 8 6 8	41068886466664040456486458447546458408266666668486844 4155445655554464456456454458645446544658755554548645844	4 3 6 4 4 4 4 6 6 4 6 6 4 6 6 4 6 6 8 4 6 6 4 4 2 4 8 6 6 2 8 4 5 4 6 8 4 6 2 2 2 5 4 6 6 4 6 2 8 4 5 6 4 6 6 2 4 5 6 6 6 6 6 6 6 6 6 6 6 6 7 8 6 6 7 8 6 7 8 7 8	726665 5767	17 1845 19 1846 1846 1846 1846 1846 1846 1846 1846	5.000 5.005 5.015 5.020 5.025 5.030 5.045 5.055 5.050 5.055 5.065 5.075 5.080 5.055 5.080 5.055 5.065 5.075 5.080 5.055 5.090 5.105 5.115 5.	486 566 400 644 644 648 632 649 649 649 649 649 649 649 649	6448884848484644868840044788464864407745888786846656465644664668646686466864	5888446 24486 44048 4466 245 284440888 24866 466 486 48 46 52 46 46 46 46 46 52 46 5	724 786 772 7726 772 7726 772 7726 772 7726 772 772	31611032 1 1 1 2 2 9 1 1 1 1 2 1 3 1 3 1 1 1 2 2 1 1 1 1 1 2 2 1 1 1 1

Lead	Gear on Worm	ıst Intermediate	2nd Intermediate	Gear on Screw	Angle	Lead	Gear on Worm	1st Intermediate	2nd Intermediate	Gear on Screw	Angle	Lead	Gear on Worm	ıst Intermediate	2nd Intermediate	Gear on Screw	Angle
5.250, 5.250, 5.260, 5.275, 5.260, 5.275, 5.288, 5.295, 5.295, 5.295, 5.295, 5.295, 5.333, 5.	56 14 14 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4	151641444444444514544441414544451454444446464646	26 1 1 5 6 6 2 2 3 6 8 2 8 8 1 8 2 1 1 1 1 1 1 1 1 1 0 1 8 6 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	50 86 72 64 56	31 26 23 52 27 12 20 12 20 8 24 12 25 13 25 25 27 12 20 12 20 8 24 12 25 17 7 22 25 25 25 25 25 25 25 25 25 25 25 25	5.500 5.505 5.515 5.525 5.525 5.525 5.5325 5.6325 5.73	6502 4446 276 4406 88 48 456 88 48 456 68 88 48 458 88 48 458 88 48 458 88 48 458 88 48 458 88 48 458 88 45 458 88 45 458 88 88 458 88 88 88 88 88 88 88 88 88 88 88 88 8	37268 0 448 1 7 0 44 2 4 4 4 0 0 6 4 4 4 4 5 6 6 4 4 2 0 4 4 4 4 5 6 6 7 4 6 4 6 7 4 6 7 4 4 8 6 8 4 4 4 5 6 6 7 4 6 7 4 6 7 4 6 7 4 6 7 4 6 7 4 6 7 4 6 7 4 6 7 4 6 7 4 6 7 6 7	8 5 4 2 5 2 4 2 5 2 6 4 5 5 2 5 2 6 4 5 8 4 4 5 6 4 2 2 4 4 4 5 6 4 4 2 8 6 4 4 5 8 6 4 4 5 8 6 5 6 6 5 7 6 5 2 4 5 6 4 5 8 8 4 4 5 6 6 6 6 6 5 7 6 5 7 6 5 7 6 5 7 7 8 7 8 7 8 7 8 7 8 7 8 7 8 7 8 7 8	726 646 866 66 866 66 866 866 866 866 866	458 31222 314 1711 171 171 171 171 171 171 171 171	5.755555555555555555555555555555555555	6868644886644868644486886447866874456886478446647866447866864784466864784668647846647866878477446647846646478466478466478466478466464784664646464	\$3845444448086444468080664008066450645064446644564466446644664466444664446644	3 2 4 5 6 3 2 2 4 4 4 5 6 3 4 4 7 2 4 2 6 2 2 5 5 3 2 5 5 3 2 5 6 2 5 6 4 4 4 5 6 5 5 2 2 2 2 5 5 2 2 2 2 5 5 5 5 5 5	72 56 72 72 48 72 64 72 56 64 72 72 72 72 72 72	153131111112333321331131313438704 1334333233332333321331231313131313134333133233333333
								=						_			_

PLAIN AND DIFFERENTIAL INDEXING ON BROWN & SHARPE MILLING MACHINES

The general arrangement of the universal dividing head is illustrated in Figs. 1, 2, and 3. As indicated by the diagrammatic sketch Fig. 2, the worm wheel A is secured to the main spindle of the spiral head and rotated by means of the worm shaft and single-threaded worm B. The index plate (having rows of equally spaced holes) remains stationary during the dividing operation, and is fitted with adjustable sector arms which obviate the necessity of counting the number of holes through which the index crank requires to be moved each time a division is made on the surface of the work. The stan-

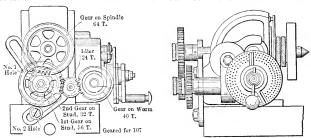


Fig. 1.—Brown & Sharpe Dividing Head Arranged for Differential Indexing

dard ratio between the worm B and the worm wheel A is 1: 40; and to find the movement of the index crank for any required division, the following formula is employed: The movement of the index crank $= \frac{40}{N}$ where N is the number of equal divisions required.

Example: Let it be required to divide the circumference of a piece of work into 48 equal parts.

The movement of the index crank for each division = $\frac{40}{48} = \frac{5}{6}$ revolutions.

An index plate having a row of 18 holes would be chosen, and the sector arms set to limit the movement of the index crank to 15 spaces for $\frac{5}{6} = \frac{15}{18}$.

GENERAL PRINCIPLE OF DIFFERENTIAL INDEXING

The number of equal divisions which may be obtained by simple indexing (with the index plates usually provided by milling-machine makers) is strictly limited, and does not meet all the requirements called for in practice.

Differential indexing provides the most convenient way of overcoming this difficulty, this method being simpler than compound indexing. In the differential system the dividing operation is performed as in simple indexing, the only difference being that the index plate instead of remaining stationary during the process of indexing, is made to move relatively to the index crank, being connected to the main spindle of the spiral head by a set of change gears, which may be arranged to give either a positive or negative movement to the

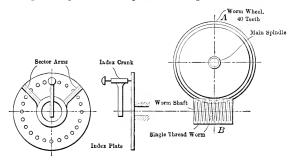


FIG. 2

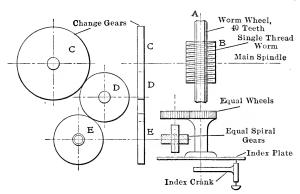


Fig. 3

index plate; whichever is found necessary to determine the actual motion which must be given to the index crank in order to satisfy the formula given above for simple indexing: Actual movement of the index crank = $\frac{40}{100}$.

The two views in Fig. 1 and the diagram, Fig. 3, will serve to give an idea of the arrangement of the gearing, which is adopted in differential indexing.

For any movement of the index crank the motion is transmitted to the index plate (which is free to rotate on the worm shaft) as follows: The index crank drives through the worm shaft and worm B to the worm wheel A, which in turn transmits the motion through the change gears, spiral gears and equal gears, the last of which is connected directly to the index plate. The last pair of gears being equal and driven through equal spiral gears, whatever number of revolutions are given to the gear E, the index plate will make the same number. It is therefore convenient to consider the revolutions of the gear E, as the revolutions of the index plate in all calculations.

To illustrate the influence of the gearing on the index plate and indexing operation, consider the following example: Required to

index for 107 divisions:

If we use the plate having 20 holes and move 8 holes per division, as in simple indexing for 100 divisions, 100 moves will of course be required to rotate the worm 40 turns, which in turn rotates the spindle once. If now we make 107 moves with the index plate fixed as in simple indexing, we will obtain $107 \times \frac{8}{20} = 42.8$ revolutions of

the worm, which is 2.8 in excess of what is required. Therefore the index plate must be geared so that it will move back 2.8 turns while the spindle is revolving once; that is, the ratio of the gearing must be 2.8 to 1.

$$\frac{2.8}{1} = \frac{2.8}{2} \times \frac{2}{1}$$

$$\frac{2.8}{2} \times \frac{20}{20} = \frac{56}{40}$$

$$\frac{2}{1} \times \frac{32}{32} = \frac{64}{32}$$

Then $\frac{2.8}{1} = \frac{56}{40} \times \frac{64}{3^2}$ and the gears will be 64 and 56 for the spindle and first gear on stud, and 40 and 32 for the worm and second gear on stud, as shown in Fig. 1. As compound gears are used, but one idler is required to cause the index plate to move in a direction opposite to that of the crank. For this purpose an idler having 24 teeth is employed.

Formula for Finding the Gear Ratio

A simple formula for the determination of the gear ratio necessary to rotate the index plate as required for any given number of teeth is derived as follows:

Let N equal the number of divisions required to be indexed.

Let n equal some number either greater or smaller than N, which can be obtained directly by simple indexing.

Let $\frac{40}{n}$ equal the index setting; that is the setting of the sector arms for each movement of the index crank;

Then $n - N \times \frac{40}{n}$ equals the gear ratio.

If the number chosen for n is greater than the number of divisions required (N) the index plate must be geared to have a positive motion, that is to rotate in the same direction as the index crank. If the number n is less than N the index plate is geared to have a negative motion, that is, to rotate in opposite direction to the crank.

Application of the Formula

Suppose we wish to obtain 63 divisions: choose any number for n which may be obtained by simple indexing, say 60, then

$$(n - N) = (60 - 63) = -3$$

This number (-3) when multiplied by the value of the index setting will give the gear ratio. The index setting equals $\frac{40}{41}$, equals $\frac{40}{62}$, then

$$(n-N)\frac{40}{n} = -3 \times \frac{40}{60} = -\frac{60}{30}$$
 or $\frac{2}{1}$ as the gear ratio.

 $(n-N)\frac{40}{n}=-3\times\frac{40}{60}=-\frac{60}{30}$ or $\frac{2}{1}$ as the gear ratio. We can therefore use gears of 48 and 24 teeth, the 48 gear being the driver and the 24 gear the follower; that is, the 48 gear being on the spindle and the 24 gear on the worm. As n is smaller than Nthe idlers are arranged to give a negative movement to the index plate.

The index setting is found above as $\frac{40}{n}$ which equals $\frac{40}{60}$ or $\frac{2}{3}$. We can thus use the 39 hole circle in the index plate and set the sector for 26 holes, this giving the setting as $\frac{26}{39}$ or $\frac{2}{3}$; that is, we set the sector and index pin exactly the same as for simple indexing of 60 divisions.

The tables on the following pages give the dividing head gears for indexing all numbers up to 730.

ions				No. 1	Hole		Idl	ers
Number of Divisions	Index Circle	No. of Turns of Index	Gear on Worm	First Gear on Stud	Second Gear on Stud	Gear on Spindle	No. 1 Hole	No. 2 Hole
2 3 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26 27 28 29 30 31 32 33 34 35 36 37 38 38 39 39 40 40 40 40 40 40 40 40 40 40	Any 39 Any 39 Any 49 Any 27 Any 33 39 49 39 20 17 27 19 Any 21 33 23 39 20 39 27 49 29 39 31 20 33 17 49 27 37 19 39 Any 41 21	10 10 8 6 5 5 4 4 4 3 3 3 3 2 2 2 2 2 2 1 1 1 1 1 1 1 1 1 1						

sions				No. 1			Idl	ers
Number of Divisions	Index Circle	No. of Turns of Index	Gear on Worm	First Gear on Stud	Second Gear on Stud	Gear on Spindle	No. 1 Hole	No. 2 Hole
43 44 45 46 47 48 49 50 51 52 53 54 55 55 55 57 58 58 66 67 68 69 70 77 77 77 78 78 78 78 78 78 78 78 78 78	43 33 27 23 47 18 49 20 17 39 49	Osado 4)- obade especia de comencia de com	24 56	40	24	48 72	24	44
55 56 57 58 59	33 49 21 29 39 39	ଖ୍ଞାନାର ଜୀକ ପ୍ରାପ ହାର ଜାନ ଖ୍ଞାନାର ଜୀକ ପ୍ରଥମ ହାର ଜାନ	56 48			40 32	24 44	44
61	39 31	2 6 3 9 2 0	48			32	24	44
63 64 65	39 16	32 30 64 30	24			48	24	44
67	39 33 21	33 12 21	28			48	44	
69	17 20	17 12 20 28	40			56	24	44
70 71	49 18	10 18 15	72			40	24	
72 73 74	27 21 37 15	27 121 20 37 8	28			48	24	44
76 77 78	19 20 39 20	10990009 12239	32			48	44	
79 80	20	$\begin{array}{c c} \frac{1}{2} \frac{0}{0} \\ \frac{1}{2} \frac{0}{0} \\ \end{array}$	48			24	44	
81 82	20 41	$\begin{array}{c c} \frac{10}{20} \\ \frac{20}{41} \end{array}$	48			2.1	24	44
83	20	$\frac{1}{2}\frac{0}{0}$	32			48	24	44

ions				No. 1	Hole		Idle	ers
Number of Divisions	Index Circle	No. of Turns of Index	Gear on Worm	First Gear on Stud	Second Gear on Stud	Gear on Spindle	No. 1 Hole	No. 2 Hole
84 85 86 87 88 89	21 17 43 15 33 18	OH. NO.08, NO.51% (NO.00 K.00.51%) = NO.000 K.00 K.00 K.00 K.00 K.00 K.00 K.00	40 72			24 32	24 44	44
91	27 39	$\frac{\frac{1}{2}}{\frac{1}{3}}\frac{8}{9}$	24			48	24	44
92 93 94	39 23 18 47	$ \begin{array}{c c} \frac{10}{23} \\ \frac{8}{18} \\ \frac{20}{47} \end{array} $	24			32	2.1	44
95 96 97 98	19 21 20 49	$\begin{array}{c} \frac{8}{19} \\ \frac{9}{21} \\ \frac{8}{20} \\ \frac{20}{20} \end{array}$	28 40			32 48	24 44	44
99 100	20	4 9 2 0 8	56	28	40	32		
101 102 103 104	20 20 20 39	20 8 20 8 20 8 20 8 20 15 39	72 40 40	24	40	*48 32 48	24 24	24 44 44
105 106 107	43 20	S 2 T 1 6 4 3 8 2 0	86 40	24 56	24 32	48 64		24
108	27 16	1 6 1 6 1 2	32			28	24	44
110 111 112 113 114 115 116	33 39 39 39 39 23	15 30 30 30 30 30 30 30 30 30 30 30 30 30	24 24 24 24 24			72 64 56 48	32 44 44 44	
116 117 118 119	29 39 39 39 39 39	100330930930 133930930 133930	24 48 72			24 32 24	56 44 44	
121 122 123 124	39 39 39 39 31	39 139 139 139 139 131	72 48 24			24 32 24	24 24 24	44 44 44

sions				No. 1	Hole		Idl	ers
Number of Divisions	Index Circle	No. of Turns of Index	Gcar on Worm	First Gear on Stud	Second Gear on Stud	Gear on Spindle	No. 1 Hole	No. 2 Hole
125 126 127 128	39 39 39 16	191999999 191919999	24 24 24			40 48 56	24 24 24	44 44 44
129	39	1 6 1 3 3 9	24			72	24	44
130 131 132	39 20 33	$\frac{3}{9}$ $\frac{6}{20}$ $\frac{10}{10}$	40			28	44	
133 134 135	2I 2I 27	$ \begin{array}{c} 3 & 3 \\ 6 & 1 \\ 2 & 1 \\ 8 & 2 & 7 \end{array} $	2.1 28			48 48	44 44	
136 137 138 139	17 21 21 21	$ \begin{array}{r} \hline 17 \\ \underline{6} \\ \hline 21 \\ \underline{6} \\ \hline 21 \\ \underline{6} \\ \hline 14 \end{array} $	28 56 56	32	48	24 32 24	56 44	
140 141 142 143	49 18 21 21 18	19 18 18 21 61 61	48 56 28			40 32 24	44 24 24	44 · 44
144 145 146 147 148	29 21 21 37	18 29 21 61 10	28 24			48 48	24 24	44 44
149	2 [$\frac{3.7}{6}$	28			72	24	44
150	15 20	15 5 20	32			72	44	
152 153 154 155	19 20 20 31	19 20 20 20 8 31	32 32			56 48	44 44	
155 156 157 158 159 160	39 20 20 20 20	100 350 250 250 250 250	32 48 64	32	56	24 24 28	56 44	
161 162 163 164 165	20 20 20 20 41 33	변화 제외 전 전 전 전 전 전 전 전 전 전 전 전 전 전 전 전 전 전	64 48 32	32	56	28 24 24	24 24	24 44 44

sions				No. 1	Hole		I dl	ers
Number of Divisions	Index Circle	No. of Turns of Index	Gear on Worm	First Gear on Stud	Second Gear on Stud	Gear on Spindle	No. 1 Hole	No. 2 Hole
166 167 168	20	5 20 5 20 5	32 32			48 56	24 24	44 44
169	2 I 20	$\frac{\overline{2}}{\overline{2}}$	32			72	24	44
170	17 21	17 5 21	56			40	24	44
172 173 174 175 176	43 18 18 18 18 18 18 18 18	18 18 18 18 4 18	72 24 72 72	56 40 24	32 32 24	64 32 64 64	56	
177 178 170	18 18 18	1 8 1 8 4 1 8 4 1 8 4 1 8	72 72 72 72	24	48	48 32 32	24 44	
180 181 182 183 184	18	18 18 4 18 4 18 23	72 72 48	24	48	32 32 32	24 24	24 44 44
185 186 187 188	37 18 18	8 37 4 18 4 18 10	48 72	48	24	64 56	24	44 24
189 190	47 18 19	18 4	32			64	24	44
191 192 193 194	20 20 20 20 20	5275562554554564546454645454545454545454545	40 40 40 40			72 64 56 48	24 44 44 44	
196 197 198 199	49 20 20 20 20	4 20 4 20 4 20 4 20 4	40 56 100	28 40	40 64	24 32 32	56	
201 202 203 204	20 20 20	20 40 20 40 20 40 20 40 20	72 72 40 40	24 24	40	24 48 24 32	24 24	24 24 44 44
205 206	41 20	4 20	40			48	24	44

	f							
sions				No. 1	Hole		Idl	ers
Number of Divisions	Index Circle	No. of Turns of Index	Gear on Worm	First Gear on Stud	Second Gear on Stud	Gear on Spindle	No. 1 Hole	No. 2 Hole
207 208 209 210	20 20 20 21	2 4 5 6 4 5 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6	40 40 40			56 64 72	24 24 24	44 44 44
211 212 213 214	16 43 27 20	21 16 8 43 57 4	64 86 72 40	24 56	24 32	28 48 40 64	44	24
215 216 217 218 219	43 27 21 16 21	8 43 5 7 4 21 3 16 5 7	48 64 28			64 56 48	24 24 24	44 44 44
220 221 222 223	33 17 18 43 18	$\begin{array}{c} 21 \\ \frac{6}{33} \\ \frac{3}{3} \\ 17 \\ \frac{3}{18} \\ \frac{8}{43} \\ \end{array}$	24 24 86	48	24	24 72 64 64	56 44	24
224 225 226 227	27 18	3 15 27 3 18 8 49	24 24 24 56	64	28	64 40 56 72 48	44 24 44	44
228 229 230 231	49 18 18 23 18	18 3 18 4 23 3	24 24 32			48 44 48	44 48 44	
232 233 234	29 18 18	15 29 3 18 3 18 8	48 24			56 24	44 56	
235 236 237 238 239	47 18 18 18	47 3 18 3 18 3 18 3 18 3 18	48 48 72 72	2.1	64	32 24 24 32	44 44 44	
240 241 242 243 244	47 18 18 18 18 18 18 18 18	# 1 2 4 0 4 0 4 0 5 6 5 6 5 6 5 6 5 6 5 6 5 6 5 6 5 6 5	72 72 64 48	24	64	32 24 32 32	24 24 24	24 44 44 44
245 246 247	49 18 18	$\frac{3}{18}$	24 48			24 56	24 24	44 44

sions				No. 1	Hole	e	Id	lers
Number of Divisions	Index Circle	No. of Turns of Index	Gear on Worm	First Gear on Stud	Second Gear on Stud	Gear on Spindle	No. 1 Hole	No. 2 Hole
248 249 250 251 252 253 254 255 256 257 258 259 260	31 18 18 18 18 18 18 18 18 49 43 21	ऽति को जीता के को जीता के को जीता के की को की	32 24 48 24 24 24 24 48 24 56 32 24	44 40 48	32 24 28	48 40 64 48 40 56 72 64 64 64 72	24 24 56 24 24 24 24 44	44 44 24 44 24 44 24 44 24
261 262 263 264	29 20 49	29 20 20 49 5	48 40 56	64 64	24 28	72 28 72	44	24
265 266 267 268 269	33 21 21 27 21 20	33 21 23 24 27 27 21 20 21	56 32 72 28 64	40 32	24	72 64 32 48 28	44 44 44	24
270 271 272 273 274 275 276 277 278 279 280	27 21 21 21 21 21 21 21 21 21 21	27 T 27 3 T 3 T 3 T 3 T 4 7 7 1	56 56 24 56 56 56 56 56 56	32	48	72 64 24 48 40 32 24 24 21	24 24 56 41 44 44 44	44
281 282 283 284 285 286 287 288	49 21 43 21 21 21 21 21 21	49 21 43 24 31 23 23 23 23 23 23 23	72 86 56 56 56 56 24 28	24 24	56 24	24 56 24 32 40 48 24 32	24 24 24 24 24 24	24 44 44 44 44 44 44

sions				No.	1 Hole		Id	llers
Number of Divisions	Index Circle	No. of Turns of Index	Gear on Worm	First Gear on Stud	Second Gear on Stud	Gear on Spindle	No. 1 Hole	No. 2 Hole
289	21	$\frac{3}{21}$	56			72	24	44
290 291 292 293 294 295 296	29 15 21 15 21 15 37	29 15 3 21 25 15 32 15 32 1 25 15 35 35 25 15 35 35 35 35 35 35 35 35 35 35 35 35 35	40 28 48 24 48	32	40	48 48 56 48 3 ²	44 21 24 44	44
297 298 299 300	37 33 21 23 15	37 43 33 21 3 23 23 23	28 28 24	48	24	56 72 24	24 56	44
301 302 303 304	43 16 15	$ \begin{array}{c} 15 \\ 6 \\ 43 \\ 2 \\ 16 \\ 25 \\ 16 \\ 2 \\ 2 \end{array} $	24 32 72 24 48	2.1	40	48 72 48 48	24 24 44	24
305 306 307 308 309	15 15 15 16	15 15 2 13 2 16 2 15	40 72 32 40	48	40	32 32 56 48 48	24 24 44 24	44 44 24 44
311	31 16	31 16 5	64	24	24	72		
312 313 314 315 316 317 318 319 320	39 16 16 16 16 16 16 29	© 1 + 5 > 10 € 1 - 5 - 11 - 5 + 5 € 5 1 - 5 5	3 ² 3 ² 64 64 64 56 48	28 64	48 24	28 24 40 32 24 24 72	56 56 24 44 44	24
321 322 323 324 325 326 327 328	16 23 16 16 16 16	16 16 3 22 16 26 16 26 16 26 16 26 16 26 16 26 16 26 16 26 16 26 16 26 16 26 16 26 26 26 26 26 26 26 26 26 26 26 26 26	7 ² 3 ² 64 64 64 3 ² 3 ²	24	64	24 64 24 32 40 24 28	24 24 24 24 24 24	24 44 44 44 44 44 44
329	41 16	5 41 2 16	64	24	24	72		24

sions				No.	ı Hole		Id	lers
Number of Divisions	Index Circle	No. of Turns of Index	Gear on Worm	First Gear on Stud	Second Gear on Stud	Gear on Spindle	No. 1 Hole	No. 2 Hole
330 331 332 333 334 335 336 337 338 340 341 342 343 344 345 346 347 350 351 352 353 354 355 356 357 358 359 360 361 362 363 364 365 366 366 366 366 366 366 366 366 366	33 16 16 18 16 18 16 18 17 43 18 18 18 18 18 18 18 18 18 18	\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\	64 32 24 32 72 32 86 32 24 86 32 40 24 72 86 24 72 72 72 72 72 72 72 72 72 72	44 48 40 24 64 56 24 44 40 24 32 48 28	24 44 32 24 32 24 32 24 32 24 32 24 32 24	48 48 48 72 56 40 64 56 40 64 64 40 64 40 64 40 64 40 40 40 40 40 40 40 40 40 4	24 44 24 24 44 45 56 56 56 24 24 24 24 44 44 24 24 24 24 24	24 44 44 24 44 44 24 24 24 24 24
369 370	41 37	$\frac{18}{41}$ $\frac{4}{37}$	72 32	²⁴ 56	24 28	64		24

suc				No. 1	Hole		Idl	ers
Number of Divisions	Index Circle	No. of Turns of Index	Gear on Worm	First Gear on Stud	Second Gear on Stud	Gear on Spindle	No. 1 Hole	No. 2 Hole
371 372 373 374 375	21 18 20 18 18	$\begin{array}{c} \frac{2}{21} \\ \frac{2}{18} \\ \frac{2}{20} \\ \frac{2}{18} \\ \frac{2}{18} \end{array}$	32 48 40 72 24	56 48 64	24 32 32	64 64 72 56 40	24	44 24 44
376 377 378 379 380	47 29 18 20	1	24 32 48	56	40	24 64 72	56 24	44
381 382 383 384 385 386 387 388 388	18 20 20 20 20 20 20 43 20 20	~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~	24 40 40 40 32 40 32 40 40	56	28	56 72 68 1 64 48 56 64 48	24 24 44 44 44 44 44 56	44
390 391 392	39 20 49	$ \begin{array}{r} \frac{4}{39} \\ \frac{2}{20} \\ \underline{5} \end{array} $	48	24	40	72		
393 394 395 396 397 398 399	20 20 20 20 20 20 20 21	4 9 2 0 2 0 2 0 2 0 2 0 2 1 0 2 1 0 2 1 0 0 0 0	40 40 64 56 64 100 32	28 24 40	40 40 64	28 24 32 32 32 32 32 64	44 56 44	
400 401 402 403 404	20 21 21 20 20	$ \begin{array}{c c} \hline 2 0 \\ \hline 2 1 \\ \hline 2 1 \\ \hline 2 2 1 \\ \hline 2 0 \\ \hline 2 0 \\ \hline 2 0 \end{array} $	56 28 64 72 64	32 24 24	24 40 40	76 ¹ 48 3 ² 48	44	24 24
405 406 407 408 409 410	20 20 20 20 20	$\begin{bmatrix} \frac{2}{20} \\ \frac{2}{20} \\ \frac{2}{20} \\ \frac{2}{20} \\ \frac{2}{20} \\ \frac{2}{20} \\ \frac{2}{4} \end{bmatrix}$	64 40 40 40 40	24	32	32 24 28 32 48	24 24 24 24 24	44 44 44 44 24
+10	41	4 T						

Note. Special gears in this and following tables are 46, 47, 52, 58, 68, 70, 76, 84.

ions				No. 1	Hole Hole		Idl	ers
Number of Divisions	Index Circle	No. of Turns of Index	Gear on Worm	First Gear on Stud	Second Gear on Stud	Gear on Spindle	No. 1 Hole	No. 2 Hole
411 412 413 414 415 416 417 418	21 20 21 21 20 20 21 20	$\begin{array}{c} 2 \\ 2 \\ 1 \\ 2 \\ 0 \\ 2 \\ 1 \\ 2 \\ 1 \\ 2 \\ 1 \\ 2 \\ 2 \\ 0 \\ 2 \\ 2 \\ 2 \\ 2 \\ 2 \\ 2 \\ 2$	28 40 48 56 32 40 56 40	32	48	24 48 32 32 48 64 24 72	56 24 44 44 24 24 24	44 44 44
419 420 421 422 423 424 425 426 427 428 429	33 21 20 20 21 43 21 21 20 20 21	200333212022021432112022021	44 48 40 72 86 72 56 40 40 28	28 56 44 24 24 48 48 56	24 40 32 56 24 56 32 32	72 64 48 48 40 32 72 64 24	24	24 24 24 24 44 24 24 44
430 431 432 433 434 435 436 437 438 439	43 21 20 20 21 21 20 23 21 43	200221433211220022143322214332333222443333333333	72 40 40 48 28 40 32 28 86	44 56 44 48	28 28 24 24	48 64 72 64 40 72 64 48 72	24 24 44 24	24 24 24 44 44 24
440 441 . 442 . 443 444 445 446 447 448 449 450	33 21 20 20 21 33 33 21 20 33 33	20021 1 1 2 2 3 3 2 1 4 3 3 3 2 1 2 2 2 2 4 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3	32 40 40 56 64 44 28 40 64 44	56 48 48 32 64 32	24 24 24 44 44	64 72 86 64 40 24 72 72 72 40	24 24 24 24	44 24 24 24 24 48 44 24 24 32

sions				No. 1	Hole		Idl	ers
Number of Divisions	Index Circle	No. of Turns of Index	Gear on Worm	First Gear on Stud	Second Gear on Stud	Gear on Spindle	No. 1 Hole	No. 2 Hole
451 452 453 454 455 456 457 458 459	33 33 33 49 49 21 33 33 27	3 3 3 4 9 4 9 1 3 8 3 2 7 6 8 3 8 2 7 6 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8	24 44 44 56 28 56 44 44 44	64 40 64	28 32 24	24 48 52 1 72 64 72 68 1 72 72	2.1 2.1 2.1 2.1 2.4 2.4	44 40 40 24 40 24
460 461 462 463 464 465 460 467 468 469	23 33 33 21 33 33 49 33 39 49	23 33 33 32 2 2 33 33 33 44 9 33 33 34 45	44 32 56 44 44 56 44 28 28	28 64 48 24 48 48 48	24 28 24 28 24 28 32 24	72 64 86 56 100 64 72 56 48	2.1	24 44 21 24 24 24
470 471 472 473 474 475 476 477 478 479 480	47 49 49 33 49 49 49 27 49 49 49	\$\text{Solution } \text{Solution } \text	56 56 48 56 56 56 24 56 56 56	32 32 64 32 40 48 24 32 32	28 28 32 28 28 28 24 28 28 28	76 ¹ 72 72 64 48 64 56 64 44 40	24	24
481 482 483 484 485 486 487 488	37 33 49	$ \begin{array}{r} 49 \\ 3 \\ \hline 4 \\ \hline 49 \\ \hline 4 \end{array} $	24 · 44 56	56	24 28	24 72 32	56 44	24
485 486 487	49 23 27 39	4 9 2 3 2 7 2 7 3	56 46 ¹ 3 ² 24	24 24 56 72	28 24 28 52 1	32 100 64 44		24
488 489 490	39 33 23 49	$ \begin{array}{ c c c } \hline 3 9 \\ 3 3 3 \\ \hline 2 \\ \hline 2 \\ \hline 4 9 \end{array} $	44 46 ¹	64 58 ¹	32 24 32	72 64		24 24

¹ Special gear.

No. 1 Hole Idlers Idlers			ſ	l	1		1		
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	sions				No. 1	Hole		Id	lers
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	Number of Divi	Index Circle	No. of Turns of Index	Gear on Worm	First Gear on Stud	1 7	Gear on Spindle	No. 1 Hole	No. 2 Hole
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	492 493 494 495 496 497 498 499 500 501 502 503 504 505 506 507 508 510 511 512 513 514 515 516 517 518 519 521 522 523 524 525 526	41 29 39 27 49 49 49 49 49 49 49 49 49 49 49 49 27 49 27 49 27 49 27 49 27 49 27 49 27 49 27 49 27 49 49 49 49 49 49 49 49 49 49 49 49 49	\$\text{8}\$\text{9}\$\text{9}\$\text{7}\$\text{9}\$\t	32 32 32 56 56 48 56 56 56 56 56 56 56 56 56 24 56 56 28 56 72 32 48 72 72 72 72 56 32	64 40 24 56 24 32 32 32 64 40 32 32 40 44 48 32 56 48 56 76 48 68 1 32 40 40 40 40 40 40 40 40 40 40	24 24 24 28 28 28 28 28 28 28 28 28 28 28 28 28	56 72 64 64 32 32 64 48 40 44 48 64 48 64 48 64 64 64 64 64 64 64 64 64 64	24 24 56 24 44	24 44 24 24 24 24 24 24 24 24 24 24 24 2

¹ Special gear.

sions				No. 1	Hole		Id	ers
Number of Divisions	Index Circle	No. of Turns of Index	Gear on Worm	First Gear on Stud	Second Gear on Stud	Gear on Spindle	No. 1 Hole	No. 2 Hole
531 532 533 534 535 536 537 538 549 541 542 543 544 545 546 547 550 551 552 553 553 553 553 553 553 553 553 553	27 27 27 27 27 29 49 27 39 27 15 39 27 27 49 27 15 15 27 49 27 27 49 27 49 27 49 27 15 15 15 15 15 15 15 15 15 15 15 15 16 17 17 18 18 18 18 18 18 18 18 18 18 18 18 18	~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~	72 72 72 72 72 72 72 58 72 58 72 72 40 32 32 72 72 32 32 72 72 32 32 72 72 72 32 32 72 72 32 32 72 40 48 60 72 72 72 72 34 40 48 60 72 72 72 72 72 72 72 72 72 72 72 72 72	32 32 32 28 56 48 56 44 24 56 44 32 32 40 56 44 32 40 56 44 32 40 56 44 32 40 56	48 48 48 56 24 24 32 32 48 24 24 48 40 24 24 24 24 24 24 24 24	48 64 56 32 40 64 32 72 56 48 64 64 64 64 64 64 64 64 64 64	24 44 24 24 44 44 44 44 24 24 44	44 24 24 24 24 24 24 24 24 24 24 24 24

¹ Special gear.

ions				No. 1	Hole		Idle	ers
Number of Divisions	Index Circle	No. of Turns of Index	Gear on Worm	First Gear on Stud	Second Gear Stud	Gear on Spindle	No. 1 Hole	No. 2 Hole
571 572 573 574 575 576 577 578	43 15 15 41 15 15 43 15 15 29	3 15 15 15 15 3 11 15 15 15 15 2 3 17 15 15 2 17 17 17 17 17 17 17 17 17 17 17 17 17	86 40 40 32 24 40 86 48	28 28 32 44	64 24 64 40	32 64 72 64 40 64 44 64 56	24 24 44 24	44
579 580 581 582 583 584 585	15 15 27 15 15	15 15 15 43 17 27 17 17 17 17 17 17 17 17 17 17 17 17 17	48 40 72 48 24 72	32 64 32 48	40 24 40	76 ¹ 48 86 64 24 56 28	44 56	24
585 586 587 588 589 590	29 15 15 15 15	15 15 15 15 15 15 15 15	72 58 1 40 72 48 40	44	40	28 32 48 32 24	24 44 44 44	44
592 593 594 595 596	16 15 33 15	1, 0 1, 0 1, 0 1, 0 1, 0 1, 0 1, 0 1, 0	24 72 32 72 72	28 56	40 28	72 48 64 24 32	44	
597 598 599 600	33 16 43 15	3 3 1 1 1 6 3 4 3 1 1 5 1 5	44 64 86	56 56 44	24 24 24	72 72 84		24
601 602 603 604 605	29 43 15 16 15	29 3 43 1 15 16 1	58 ¹ 32 72 32 72	56	48	72 64 24 72 24	24 24 24	24 44 24 44
606 607 608 609	15 15 16	1 5 1 7 5 1 7 5 1 7 6 1 7 5	72 72 32 40	24 28	40 40	48 48 64 24	44 24	24 24 44
610	15	15	48			32	24	44

¹ Special gear.

sions				No. 1	Hole		No. 2	Hole
Number of Divisions	Index Circle	No. of Turns of Index	Gear on Worm	First Gear on Stud	Second Gear on Stud	Gear on Spindle	No. 1 Hole	No. 2 Hole
611 612 613	15 15 16	1 15 15 15 1-	72 40 64	44	40	48 32 72	24	24 44
614 615 616	15 15 16	1 6 1 1 5 1 5 1 1 5 1 7 6	72 24 32	48 48	40	56 24	24 44	24 44
617 618 619	33 15 16	33 33 15 15 16	44 40 48	3 ² 28	32	48 86 48 72	24	44
620 621 622 623	31 15 16 16	31 1 15 16 16	40 64 64	24 24	24 24	56 72 68 1	2.4	44
624 625 626	16 15 16	16 16 15 15 16	24 24 32	·	,	24 40 28	56 24 56	44
627 628 629	16 16	15 16 1 16 1	40 32 64 64			72 24 44 40	24 56 24 24	44
630 631 632 633	16 16 16	$ \begin{array}{c} 1.6 \\ 1.6 \\ 1.6 \\ 1.6 \\ 1.6 \\ 1.6 \end{array} $	64 64 64	28	56	72 32 28	44 44	
634 635 636	16 15 16	$\frac{1}{16}$ $\frac{1}{15}$ $\frac{1}{16}$	64 24 56	28	48	24 56 24	44 24	44
634 635 636 637 638 639 640	49 29 33 16	4 9 2 9 2 9 2 3 3 1 6	24 48 44	64 28	24 32	24 72 64	56	24
641 642 643	33 16 16	-10-10-10-10-10-89-10-10-89-10-10-10-10-10-10-10-10-10-10-10-10-10-	44 72 64	32 24 28	48 64 56	76 ¹ 24 24		24 24
644 645 646 647	49 15 16 16	15 15 16	56 24 64 64			32 72 24 28	44 24 24 24	44 44 44
647 648 649 650	16 33 16	$ \begin{array}{r} \begin{array}{r} $	64 72 64			32 48 40	24 24 24 24	44

¹ Special gear.

sions				No. 1	Hole		No. 2	Hole
Number of Divisions	Index Circle	No. of Turns of Index	Gear on Worm	First Gear on Stud	Second Gear on Stud	Gear on Spindle	No. 1 Hole	No. 2 Hole
651 652 653 654	16 16 33 16	101523151616161616283661616171586918161628888887717288887161866978888716181688887171816888871718168888717181688887171818188888717181818888871718181888887171818188888717181818888871718181888887171818188888717181818888871718181888888	64 32 72 64	28	44	44 24 48 56	24 24 24	24 44 44
654 655 656 657 658	16 16 18 16	$ \begin{array}{c c} 1 \\ 1 \\ 6 \\ 1 \\ 1 \\ 8 \\ 1 \end{array} $	64 24 32 64	40 48 24	32 24 24	48 24 56	24	24 44 24
659 660	16 33 16	$ \begin{array}{c c} \hline 16\\ \hline 16\\ \hline 2\\ \hline 3,3 \end{array} $	64	24	24	72 76 ¹		24
661 662 663	16 17	16 16 16 17	64 64 24	56 44	48 24	72 48 24	56	24 24
664 665 666	16 49 18	1 8 4 9 1 1 8	32 56 24	_		48 40 72	24 24 44	44 44
667 668 669	16 16 33	$ \begin{array}{r} 1 \\ 1 \\ 6 \\ 1 \\ 1 \\ 6 \\ 2 \\ 3 \\ 3 \end{array} $	64 32 44	48	32	72 56 24	24 24	24 44 24
670 671 672	33 33 18	$\frac{\frac{2}{33}}{\frac{3}{33}}$	72 72 24	48	44	40 48 64	24 44	24 24
673 674 675 676	16 33 33 16	$ \begin{array}{r} \frac{1}{16} \\ \frac{2}{33} \\ \frac{2}{33} \end{array} $	48 72 44	44 56	32 44	72 48 40	24	24 24 24
677 678	16 18 18	$ \begin{array}{r} \frac{1}{16} \\ \frac{1}{18} \\ \frac{1}{18} \end{array} $	32 48 24 28	32	24	72 86 56	24 44	44
679 680 681	49 17 33	3 4 9 1 1 7 2	l			44 56	24 24	40 24
682 683 684	33	3 3 3 3 1 1 1 6 1 1 6 1 1	44 48 32 32			64 86 64	24 24 44	24 44
685 686	18 18 15 18	$\begin{bmatrix} \frac{\overline{18}}{18} \\ \frac{1}{18} \\ \frac{\overline{1}}{15} \\ 1 \end{bmatrix}$	24 40	56 64	48 24	40 86	48	24
687 688 689	16 16 39 18	$ \begin{array}{c c} & 1 \\ & 1 \\ & 1 \\ & 6 \\ & 2 \\ & 3 \\ & 9 \end{array} $	24 24 24	48	24	44 72 56	24	44
690	18	18	24			40	56	

¹ Special gear.

sions				No. 1	Hole		Id	llers
Number of Divisions	Index Circle	No. of Turns of Index	Gear on Worm	First Gear on Stud	Second Gear on Stud	Gear on Spindle	No. 1 Hole	No. 2 Hole
691 692 693 694	18 18 18	$\begin{array}{c c} \frac{1}{18} \\ \frac{1}{18} \\ \frac{1}{18} \\ \frac{1}{17} \\ \frac{1}{18} \end{array}$	48 72 32 68 1	32 56	24 32	58 1 64 48 56 100	44 24	44
694 695 696 697 698	17 18 18 17 18 18	18 17	72 24 24 72	24	24	24	56 24	44
699 700 701 702	17 18	1 1 8 1 7 1 7 1 1 8 1 1 1 8 1 1 8 1 1 8 1 1 8 1 1 8 1 1 8 1 1 8 1 1 8 1 1 8 1 1 8 1	72 48 72 68 1 24	40 48	32 32	48 56 64 56 24 72	44 56	24
703 704 705 706 707	18 18 18 18	1 1 8 1 1 8	24 72 48 72 72	24	24	64 40 56	44 44 24 24	
708 709 710 711	19 18 18 18 18 18 18 18 18 18 18 18	18 8 17 18 18 17 18 18 17 18 18 17 18 18 17 17 18 18 17 17 18 18 17 17 18 18 17 17 18 18 17 17 17 17 18 18 17 17 17 17 17 17 18 18 17 17 17 17 17 17 17 17 17 17 17 17 17	72 72 72 64			48 44 40 32	24 24 24 44	
712 713 714 715 716	18 18 18	$ \begin{array}{r} \hline 18 \\ 18 \\ \hline 18 \\ 18 \\ \hline 18 \\ 18 \\ \hline 18 \\ 18 \\ \hline 18 \\ 18 \\ \hline 18 \\ \hline 18 \\ $	72 72 72 72 72 72	3 ² 28	64 56 64	32 28 24 40 32	24 44 44	
717 718 719 720	18 33 17 18	$\frac{1}{18}$ $\frac{2}{33}$ $\frac{1}{17}$	72 44 68 ¹	58 1 52 1	64 24 24	32 64 72		24 24
721 722 723 724	21	$\frac{1}{21}$ $\frac{1}{19}$ $\frac{1}{18}$	24 32 72 72	64 24 28	3 ² 64 56	68 ¹ 64 3 ² 3 ²	44	24 24
725 726 727	19 18 18 18 18 18	18 18 18 18 18	72 72 72	24	56 48	40 24 28	24 24	24 44 44
728 729 730	18 18 20	$ \begin{array}{r} $	72 64 32	48	24	32 32 56	24 24	44 44

¹ Special gear.

TABLE FOR INDEXING ANGLES — INDEX PLATES (See Explanation on page 191)

Holes	15	91	17	81	19	20	21	23	27	29	31	33	37	39	I	43	4	46	Holes
1-			0											1					
0	9990	0.0625	0.0558	0.0555	0.0520	0.0500	0.0666 0.0625 0.0535 0.0555 0.0526 0.0500 0.0476 0.0435 0.0345 0.0323 0.0379 0.0270 0.0270 0.0244 0.0233 0.0204	0.0.135	0.0370	0.0345	0.0323	0.0303	0.0270	0.0250	0.02 14	0.0233	0.0213	5.0204	н
0	1333	0.1250	9/11/0	0.1111	0.1053	0.1000	0.1333 0.1250 0.1176 0.1111 0.1053 0.1000 0.0952 0.0870 0.0741 0.0690 0.0645 0.0606 0.0541 0.0513 0.0488 0.0465 0.0426 0.0408	2.0870	5.074I	0090.0	0.0645	0000.0	0.0541	0.0513	0.0.188	9950.0	0.0426	5.0408	73
0	.2000	5.1875	5921.0	0.1666	0.1579	0.1500	0.2000 0.1875 0.1765 0.1666 0.1579 0.1500 0.1429 0.1304 0.1111 0.1034 0.0968 0.0909 0.0811 0.0769 0.0732 0.0698 0.0638	5.1304 C	0.1111	D. IO34	8960.0	0.0000	0.0811	09200	0.0732	8690.0	0.0638	0.0612	3
0	0.2667	5.2500	5.2353	0.2222	0.2105	0.2000	0.2500 0.2353 0.2222 0.2105 0.2000 0.1004 0.1739 0.1481 0.1379 0.1290 0.1212 0.1081 0.1026 0.0976 0.0930 0.0851 0.0816	5.I739 c	5.1481 c	0.I379	0021.0	0.1212	0.1081	0.1026	92600	0.0030	0.0851	9180.0	4
0	.3333	5.3125	2.294I	0.2777	0.2632	0.2500	0.3333 0.3125 0.2911 0.2777 0.2632 0.2500 0.2334 0.2174 0.1852 0.1724 0.1613 0.1515 0.1351 0.1252 0.1220 0.1163 0.1064 0.1020	5.2174 C	5.1852 c	0.1724	0.1613	0.1515	0.1351	0.1282	0.1220	0.1163	0.1064	o.ro20	Ŋ
	4000	2.3750	0.3529	0.3333	0.3158	0.3000	0.4000 (0.3750 0.3529 (0.3333) 0.3158 (0.3000 0.2857 (0.2600) 0.2222 (0.2609 0.1935 (0.1818 0.1622 (0.1838 0.1463 0.1305 (0.1377 (0.1224	5.2600 c	D. 2222 C	0.2069	0.1935	0.1818	0.1622	0.1538	0.1463	0.1305	0.1277	0.1224	9
0	0.4667	5.4375	0.4118	0.3888	0.3684	0.3500	0.4375 0.4118 0.3888 0.3684 0.3500 0.3333 0.3043 0.2503 0.2414 0.2258 0.2121 0.1892 0.1795 0.1707 0.1628 0.1429 0.1429	5.3043	5.2593	0.2414	0.2258	0.2121	0.1892	0.1795	0.1707	0.1628	081.1.0	0.I.129	7
0_	.5333	5,5000	0.4706	0.4444	0.4211	0.4000	0.5333 0.5000 0.4706 0.4444 0.4211 0.4000 0.3810 0.3478 0.2963 0.2759 0.2581 0.2424 0.2162 0.2051 0.1951 0.1950 0.1702 0.1053	5.3478 c	5.2963	0.2759	0.2581	0.2424	0.2162	0.2051	0.1951	0.1860	0.1702	5.1633	×
0	,6000	5625	0.5294	0.5000	0.4737	0.4500	0.6000 0.5625 0.5294 0.5000 0.4737 0.4500 0.4286 0.3913 0.3333 0.3103 0.2003 0.2777 0.2132 0.2338 0.2195 0.1915 0.1837	5.39T3	0.3333	0.3103	0.2003	0.2727	0.2.132	0.2308	0.2195	0.2093	0.1015	0.1837	6
0	9999	5.6250	0.5882	0.5555	0.5263	0.5000	0.6666 0.6250 0.5882 0.5555 0.5363 0.5000 0.4762 0.4318 0.3703 0.3448 0.3226 0.3030 0.2703 0.2564 0.2439 0.2326 0.2128 0.2040	5.43 t8 c	0.3703	0.3448	0.3226	0.3030	0.2703	0.2564	0.2439	0.2326	0.2128	0.20.40	01
0	.7333	5.6875	0.647I	0.6111	0.5789	0.5500	0.7333 0.6875 0.6471 0.6111 0.5789 0.5550 0.5238 0.4734 0.4774 0.3793 0.3548 0.3333 0.2073 0.2013 0.2583 0.2583 0.2545	5.4783 c	5.457.4	0.3793	0.3548	0.3333	0.2973	0.2821	0.2683	0.2558	0.23.10	0.2245	11
0	.8000	0.7500	0.7059	0.6666	0.6316	0.6000	0.8000 0.7500 0.7050 0.6066 0.6316 0.6000 0.5714 0.5217 0.4444 0.4438 0.3871 0.3636 0.3243 0.3247 0.2927 0.2791 0.2553 0.2449	0.5217	0.444.4	0.4138	0.3871	0.3636	0.3243	0.3077	0.2927	0.2791	0.2553	0.2419	1.2
0	3666	5.8125	0.7647	0.7222	0.6842	0.6500	0.8666 0.8125 0.7647 0.7222 0.6842 0.6500 0.6100 0.5652 0.4815 0.4483 0.4194 0.3939 0.3514 0.3333 0.3171 0.3023 0.2766 0.2653	0.5652	0.4815	0.4483	0.4194	0.3939	0.3514	0.3333	0.3171	0.3023	0.2766	0.2653	13
_ 0	.9333	5.8750	0.8230	0.7777	0.7368	0.7000	0.9333 0.8750 0.8230 0.7777 0.7368 0.7000 0.6666 0.6087 0.5185 0.4827 0.4516 0.4242 0.3784 0.3590 0.3415 0.3256 0.2979 0.2857	5.6087 c	0.5185	0.4827	0.4516	0.4242	0.3784	0.3590	0.3415	0.3256	0.2970	0.2857	14
		0.9375	0.8824	0.8333	0.7895	0.7500	0.0375 0.8824 0.8333 0.7805 0.7500 0.7143 0.6522 0.5555 0.5172 0.4839 0.4545 0.4054 0.3846 0.3659 0.3488 0.3101 0.3061	5.6522	0.5555,0	0.5172	0.4839	0.4545	0.4054	0.3846	0.3659	0.3488	0.3191	o.306r	15
	-	,	0.9412	0.8888	0.8421	0.8000	0.9412 0.8888 0.8421 0.8000 0.7619 0.6057 0.5024 0.5517 0.5161 0.4848 0.4324 0.4103 0.3902 0.3721 0.3404	5.6957 c	0.5924	0.5517	0.5161	0.4848	0.4324	0.4103	0.3902	0.3721		0.3265	91
				0.9111	0.8947	0.8500	0.9444 0.8947 0.8500 0.8 305 0.7391 0.6296 0.5862 0.5484 0.5151 0.4506 0.4359 0.4146 0.3954 0.3617 0.3409	5.7391	0.6296	0.5862	0.5484	0.5151	0.4596	0.4359	0.4146	0.3954	0.3617	0.3400	17
					0.9474	00000	0.9474 0.9000 0.8571 0.7826 0.6666 0.6207 0.5806 0.5454 0.4865 0.4615 0.4390 0.4186 0.3830 0.3673	5.7826 c	5.6666	0.6207	0.5806	0.5454	0.4865	0.4615	0.4390	0.4186	0.3830	0.3673	18
						0.9500	0.9500 0.9048 0.8261 0.7037 0.6552 0.6129 0.5757 0.5135 0.4872 0.4634 0.4419 0.4043 0.3878	5.8261 c	5.7037	0.6552	0.6129	0.5757	0.5135	0.4872	0.4634	0.4119	0.4043	0.3878	61
	_						0.9524	2.8696 c	0.7407	2.6897	0.6452	0,6060	0.5405	0.5128	0.9524 0.8696 0.7407 0.6897 0.6452 0.6060 0.5405 0.5128 0.4878 0.4651 0.4255 0.4082	0.4651	0.4255	0.4082	20
								5.9130k	2,7777	0.7241	0.6774	0.6363	0.5676	0.5385	0.9130 0.7777 0.7241 0.6774 0.6363 0.5676 0.5385 0.5122 0.4884 0.4468 0.4286	0.4884	8911.0	0.4286	2.1
-								0.9565	o.8148	0.7586	0.7097	0.6666	0.5946	0.5641	0.9565 0.8148 0.7586 0.7097 0.6666 0.5946 0.5641 0.5366 0.5116 0.4681 0.4490	0.5116	0.4681	0.4490	22
									0.8519	0.7931	0.7419	0.6969	0.6216	0.5897	0.8519 0.7931 0.7419 0.6969 0.6216 0.5897 0.5610 0.5349 0.4894 0.4694	0.5349	1684.0	p.4694	23
									2.8888°c	0.8888 0.8276 0.7742 0.7272 0.6486 0.6154 0.5854 0.5581 0.5106 0.4808	0.7742	0.7272	0.6486	0.6154	0.5854	0.5581	0.5106	0.4808	24

INDEX PLATES	27	29	31	33	37	39	14	£4	47	40	Holes
	0.9259	0.8621	0.8065	0.7575	0.6757	0.6666	0.6098	0.5814	0.0259 0.8621 0.8605 0.7575 0.0557 0.0110 0.0008 0.5814 0.5319 0.5102	0.5306	25
		0.9310	0178.0	0.8181	0.7297	0.6923	0.6585	0.6279	0.9310 0.8710 0.8181 0.7297 0.6923 0.6585 0.6279 0.5745 0.5510	0.5510	27
		0.9655	0.9032	0.8484	0.7568	0.7179	0.6829	0.6512	0.9655 0.9032 0.8484 0.7568 0.7179 0.6829 0.6512 0.5957 0.5714	0.5714	28
TABLE FOR INDEXING ANGLES			5.9677	0.0000	0.8108	0.7692	0.7317	0.6977	0.9535 0.0/07 0.7030 0.7430 0.7073 0.0744 0.0170 0.5910 0.0677 0.0000 0.8108 0.7692 0.7317 0.6977 0.6383 0.6122	0.5910	3 3
				0.9393	0.8378	0.7949	0.7561	0.7200	0.9393 0.8378 0.7949 0.7561 0.7209 0.6596 0.6326	0.6326	31
The table saves many calculations when obtaining proper				9696.0	0.8647	0.8255	0.7805	0.7442	0.9696 0.8647 0.8205 0.7805 0.7442 0.6809 0.6531	0.6531	32
index for angles, either in degrees, minutes or seconds, or all					0.8919	0.8462	0.8049	0.7674	0.8919 0.8462 0.8049 0.7674 0.7021 0.6735	0.6735	33
three collectively. An example will readily show its application.					0.9180	0.8718	0.8293	0.7907	0.9180 0.8718 0.8293 0.7907 0.7234 0.6939	0.6939	34
The angle to be indexed is one of 31 deg., 17 min., 11 sec.					0.0420	0.8974	0.8537	0.8140	0.9459 0.8974 0.8537 0.8140 0.7447 0.7143	0.7143	35
Keducing the angle to seconds = $112,031$. One turn of the					0.9730	0.0231	0.8780	0.8372	0.9730 0.9231 0.8780 0.8372 0.7760 0.7347	0.7347	36
crank = 9 deg., which reduced to seconds = $32,400$.						0.0487	0.9024	0.8605	0.0487 0.9024 0.8605 0.7872 0.7551	0.7551	37
112 621						0.9743	0.9268	0.8837	0.9743 0.9268 0.8837 0.8085 0.7755	0.7755	38
$\frac{323400}{32400} = 3.4762 \text{ turns}$							0.9512	0.9070	0.9512 0.9070 0.8298 0.7959	0.7959	39
0.546.4							0.9756	0.0302	0.9756 0.9302 0.8511 0.8163	0.8163	40
We find in the table $0.4762 = \frac{19}{21}$; thus, three turns and 10			-					0.9535	0.9535 0.8723 0.8367	0.8367	41
holes on the 21 circle give correct results to four decimal			_¥_	/~ p-~				0.9768	0.9768 0.8936 0.8571	0.8571	42
places.			7						0.9149 0.8775	0.8775	43
			\						0.0362 0.8980	0.8980	44
				_					0.9574 0.9184	0.9184	45
		_	×						0.9787,0.9388	0.9388	46
			/	\						0.9592	47
				\						0.0706	84

MILLING CUTTER, REAMER AND TAP FLUTES

THE following tables give the number of teeth or flutes suitable for milling in various types of cutters, reamers, taps, etc., and also show the forms of fluting cutters used.

END MILLS

SPIRAL TEETH

STRAIGHT TEETH

25 to 3

	,		
Dia, Mill	No. Teeth	Dia. Mill	No. Teeth
$ \begin{array}{c} \frac{1}{4} \\ \frac{4}{16} \text{ to } \frac{9}{16} \\ \frac{5}{8} \text{ to I} \\ \text{I } \frac{1}{8} \text{ to } \text{I} \frac{3}{8} \\ \text{I } \frac{1}{2} \end{array} $	6 8	$\frac{1}{4}$ to $\frac{1}{2}$	8
$\frac{5}{16}$ to $\frac{9}{16}$	8	$\frac{1}{4}$ to $\frac{1}{2}$ $\frac{9}{16}$ to $\frac{1}{16}$	10
$\frac{5}{8}$ to I	10	1 to 1 ½	12
$1\frac{1}{8}$ to $1\frac{3}{8}$	12	1 🖁 to 1 🗓	14
$1\frac{1}{2}$	14		1
Dia. Mill	No. Teeth	Dia. Cutter	No. Blades
		4	10
	1	5 6	12
			16
		7 8	18
1 ½ to 1 ½	16		20
$1\frac{5}{8}$ to $2\frac{9}{16}$	18	10	24
2\frac{5}{2} to 3	20	12	28

METAL SLITTING CUTTERS

Thickness	Pitch	Thickness	Pitch
3 ¹ 2 16 3 ² 18	3 16 1 4 9 32 5 16	35 32 36 16 1	16 112 32 8

SCREW SLOTTING CUTTERS

Cutters thinner than $\frac{1}{32}$ cut $\frac{3}{64}$ pitch. Cutters $\frac{1}{82}$ to $\frac{3}{64}$ thick cut $\frac{1}{16}$ pitch. Cutters over $\frac{3}{64}$ thick cut $\frac{1}{32}$ pitch.

PLAIN MILLING C	UTTERS	FLUTING CUTTERS
Dia. of Cutter 2 to 2\frac{3}{4} 3 to 3\frac{3}{4} 4 to 5 5\frac{1}{4} to 6 6\frac{1}{4} to 8\frac{3}{4} 9 to 9\frac{1}{4} 9\frac{1}{2} to 10 10\frac{1}{4} to 11	No. of Teeth 18 20 22 24 26 28 30 32	Form of Cutter for Milling
		Teeth in Plain Milling Cutters.

Plain cutters of $\frac{3}{4}$ -inch face and over are generally made with spiral teeth. The 12-degree angle on side of fluting cutter gives ample clearance for cutting spiral grooves with the 12-degree face set on the center line of the work.

SIDE OR STRADDLE	MILLS	
Dia. of Cutter 2 2 2 2 2 3 4 to 3 2 3 4 to 4 5 to 5 6 to 6 6 7 to 7 4 8 to 8 3 4	No. of Teeth 16 20 24 26 28 30 32 34	A
		Angular Cutter for Milling Teeth in Straddle Milling Cutters.

For milling teeth on periphery of straddle mills use angular cutter with 60 degree angle at A; for milling teeth on sides of cutters use 70°, 75° or 80° cutter according to number of teeth in cutter to be milled.

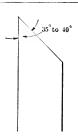
CORNER ROUNDING CUTTERS

Dia. Cutter	Rad. Circle	No. Teeth
$\begin{array}{c} 2 \\ 2\frac{1}{4} \\ 2\frac{3}{4} \text{ to } 3\frac{1}{2} \\ 3\frac{3}{4} \end{array}$	16 to 18 52 to 14 56 to 58 16 to 3	14 12 10 8

CONCAVE AND CONVEX CUTTERS

Dia. Cutter	Dia. Circle	No. Teeth
2	1/8 to 1/4	12
2 ¹ / ₄ to 3 ¹ / ₂	3/8 to 1/8	10
3 ¹ / ₂ to 4	1/4 to 2	8

FLUTING CUTTERS



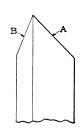
Angular Cutter for Milling Teeth in Corner Rounding, Concave and Convex Cutters.

Angular Cutters

Dia. of Cutter	No. of Teeth
2 1 2 2 3 4 3 3	18 20 22

Double Angle Cutters

Dia. of Cutter	No. of Teeth
2½ to 3	22



CUTTERS FOR SPIRAL MILLS

Dia. of Cutters	No. of Teeth
2 ½ 2 ¾ 3	18 20 22

Cutter for Milling Teeth in Angular Cutters, Double Angle Cutters and Cutters for Spiral Mills. To cut Teeth on side A of this Cutter use 60° Cutter. To cut side B use 70°-75° Cutter.

HAND TAPS		TAP FLUTING CUTTER
Dia. of Tap	No. Flutes	•
$\frac{3}{12}$ to $1\frac{3}{4}$ $1\frac{7}{8}$ to $2\frac{3}{4}$ 3 to 4 3 to 4 mm. 5 to 44 mm. 6 to 50 mm.	4 6 8 3 4 6	B
Machine Screw	TAPS	
Dia. of Tap	No. Flutes	E
No. 1 to 7 No. 8 to 30	3 4	
TAPPER TAPS		
Dia. of Tap	No. Flutes	W.
$\frac{1}{4}$ to $1\frac{5}{8}$ $1\frac{3}{4}$ to 2 $2\frac{1}{4}$	4 5 6	With 3 Flutes in Tap, B = $\frac{3}{4}$ Dia. Tap 4 Flutes in Tap, B = $\frac{1}{2}$ Dia. Tap 5 Flutes in Tap, B = $\frac{13}{8}$ Dia. Tap
MACHINE OR NUT	Taps	6 Flutes in Tap, $B = \frac{1}{32}$ Dia. Tap 7 Flutes in Tap, $B = \frac{9}{32}$ Dia. Tap
Dia. of Tap	No. Flutes	8 Flutes in Tap, $B = \frac{1}{4}$ Dia. Tap
$\begin{array}{c} {}^{3}_{16} \text{ to } {}^{5}_{16} \\ {}^{2}_{8} \text{ to } {}^{2}_{4} \\ {}^{2}^{3}_{8} \text{ to } {}^{3}_{4} \\ {}^{3}_{4} \text{ to } {}^{4} \\ \end{array}$	4 5 6 7	With 3 Flutes in Tap, $E = \frac{9}{32}$ Dia. Tap 4 Flutes in Tap, $E = \frac{1}{4}$ Dia. Tap 5 Flutes in Tap, $E = \frac{1}{64}$ Dia. Tap 6 Flutes in Tap, $E = \frac{1}{64}$ Dia. Tap
Screw Machine	TAPS	7 Flutes in Tap, $E = \frac{5}{32}$ Dia. Tap 8 Flutes in Tap, $E = \frac{9}{64}$ Dia. Tap
Dia. of Tap	No. Flutes	
$\frac{1}{4}$ to $1\frac{1}{2}$ $1\frac{9}{16}$ to 2	4 6	cutter the cutter must be central with the tap.

TAPER PIPE TAPS

TAP FLUTING CUTTERS

Dia. of Tap	No.	of Flutes				
$ \begin{array}{cccccccccccccccccccccccccccccccccccc$	8				_	85°
STRAIGHT	r Pipe	TAPS				
Dia. of Tap	No.	of Flutes			34	5 550
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$		8 9	that may	y be used i d of tap	f prefer	fluting cutter rred for fluting ace of convex
		Pipe	Новѕ			D = F
Dia. Hob	No. Flutes	Dia. Hob	No. Flutes	Dia. Hob	No. Flutes	HOB FLUTING CUTTER
18 to 14	6 8 9	$ \begin{array}{c c} & \text{I} \\ & \text{I} \stackrel{1}{4} \text{ to } & \text{I} \stackrel{1}{2} \\ & 2 \\ & 2 \stackrel{1}{2} \end{array} $	12 16 20 24	$\begin{bmatrix} 3 \\ 3\frac{1}{2} \\ 4 \\ 4\frac{1}{2} \end{bmatrix}$	28 34 36 46	50°
S	Seller	s Н овѕ		Нов Т	`APS	
Dia. Hobs	No. lutes	Dia. Hob	No. Flutes	Dia. Hob	No. Flutes	
1 to 7 1 to 7 1 to 7 1 to 7 15 to 1	6 8 10	$1\frac{5}{8}$ to $2\frac{1}{2}$ $2\frac{5}{8}$ to $2\frac{7}{8}$ 3 to 4	12 14 16	1 to 1/2 1/6 to 1/3 to 1/2 1/5 to 1/2 1/5 to 2	6 8 10 12	In fluting hobs leave land 16 inch wide on top.

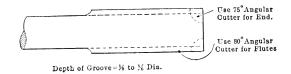
SHELL REAMERS

Dia. of Reamer	No. of Flutes	Dia. of Reamer	No. of Flutes	Dia. of Reamer	No. of Flutes
½ to 3/8	6	$\frac{21}{32}$ to $1\frac{1}{2}$	10	$2\frac{9}{32}$ to $2\frac{3}{4}$	14
$\frac{13}{32}$ to $\frac{5}{8}$	8	$1\frac{1}{3}\frac{7}{2}$ to $2\frac{1}{4}$	12	$2\frac{2}{3}\frac{5}{2}$ to 4	16

CUTTERS FOR FLUTING REAMERS									
Dia. of Reamer	R = Radius of Corner	Dia. of Reamer	R = Radius of Corner	\ .R					
1 to 3/16 1 to 3/8 1 to 5/8 1 to 1	$ \begin{array}{c} O \\ \hline 64 \\ 1 \\ 32 \\ 36 \end{array} $	$ \begin{array}{c} 1 \frac{1}{8} \text{ to } 1\frac{1}{2} \\ 1 \frac{9}{16} \text{ to } 2\frac{1}{4} \\ 2 \frac{5}{16} \text{ to } 3 \end{array} $	$ \begin{array}{c} 1 \\ 1 \\ 5 \\ 6 \\ 4 \\ 3 \\ 1 \\ 6 \end{array} $						
Dia. of Reamer	A = Am't Cut- ting Edge is Ahead of Center	Dia. of Reamer	A = Am't Cut- ting Edge is Ahead of Center						
140301010101014730 I	.011 .016 .022 .027 .033 .038 .044	11/23/4 2 14 1/2 2/4 3	.066 .076 .087 .098 .109 .120	Ja Constant					

The type of cutter shown may be used for all classes of reamers except rose reamers.

ROSE CHUCKING REAMERS



Dia. of Reamer	No. of End Cuts	No. of Flutes	Dia. of Reamer	No. of End Cuts	No. of Flutes
$\begin{array}{ccc} \frac{1}{8} & \text{to} & \frac{1}{2} \\ \frac{5}{8} & \text{to} & \text{I} \\ \text{I} & \frac{1}{8} & \text{to} & \text{I} & \frac{1}{2} \end{array}$	6 8 10	3 4 5	1 ³ / ₄ to 2 2 ¹ / ₄ to 2 ¹ / ₂ 2 ³ / ₄ to 3	12 14 16	6 7 8

TAPER REAMERS

Morse Taper		B. & S. Tapei	З .	JARNO TAPER		
No. of Taper	No. of Flutes	No. of Taper	No. of Flutes	No. of Taper	No. of Flutes	
o to 1 2 to 4 5 6 7	6 8 10 14 16	1 to 5 6 to 10 11 to 12 13 14 to 16 16 to 18	6 8 10 12 14 16	2 3 to 4 5 to 10 11 to 15 16 to 18 19 to 20	4 6 8 10 12 14	

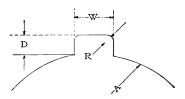
TAPER PIN REAL	1ERS	LOCOMOTIVE TAPER REAMERS		
No. of Reamer	No. of Flutes	Dia. of Reamer	No. of Flutes	
0000 to 00 0 to 7 8 to 10 11 to 14	4 6 8 10	$\begin{array}{c} \frac{1}{4} \text{ to } \frac{1}{2} \\ \frac{9}{16} \text{ to } 1\frac{1}{4} \\ 1\frac{5}{16} \text{ to } 1\frac{3}{4} \\ 1\frac{1}{16} \text{ to } 2 \end{array}$	6 8 10 12	

CENTER REAMERS

DIAMETER OF STRADDLE MILL FOR FLUTING (3 FLUTES)

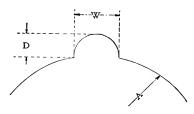
Size of Reamer	Outside Dia. of Cutter	Size of Reamer	Outside Dia. of Cutter
1" cut 8" cut ½" cut ½" cut 5" cut	2½ 2¾ 3 3 34	¾″ cut ¾″ cut 1″ cut	3½ 3¾ 4

CUTTER KEYWAYS



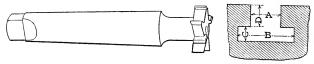
SQUARE KEYWAY

Dia. Hole, A Width Key, W Depth, D Radius, R	$\frac{\frac{3}{3}}{\frac{3}{6}}$	1 8 1 16	32 5 63	$\frac{\frac{3}{16}}{\frac{3}{32}}$	1 \frac{7}{16} - 1 \frac{3}{4} \\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\\	$ \begin{array}{c} 1\frac{1}{16}-2 \\ \frac{5}{16} \\ \frac{5}{32} \\ .060 \end{array} $	$ \begin{array}{c} 2\frac{1}{16} - 2\frac{1}{2} \\ \frac{3}{8} \\ \frac{3}{16} \\ .060 \end{array} $	$2\frac{9}{16} - 3$ $\frac{7}{16}$ $\frac{3}{16}$.060
---	-----------------------------------	-------------------	---------------	-------------------------------------	--	--	--	--



HALF-ROUND KEYWAY

Dia. Hole, A. Width, W Depth, D	1 8	$\begin{bmatrix} \frac{11}{16} - \frac{13}{16} \\ \frac{3}{16} \\ \frac{3}{32} \end{bmatrix}$	$\begin{bmatrix} \frac{7}{8} - 1 \frac{3}{16} \\ \frac{1}{4} \\ \frac{1}{8} \end{bmatrix}$	$ \begin{array}{c c} \mathbf{I} & \frac{1}{4} - \mathbf{I} & \frac{7}{16} \\ \hline & \frac{5}{16} \\ \hline & \frac{5}{32} \end{array} $	$1\frac{1}{2}-2$ $\frac{\frac{3}{8}}{\frac{3}{16}}$	$\begin{bmatrix} 2\frac{1}{16} - 2\frac{7}{16} \\ \frac{7}{16} \\ \frac{7}{32} \end{bmatrix}$	$2\frac{1}{2} - 3$
---------------------------------	-----	---	--	---	---	---	--------------------



Width of Slot A Inches	Diameter of Neck of Cutter Inches	Width of Slot B Inches	Depth C Inches	Extreme Limit D Inches
$- \nabla^{\alpha} G_{\mu}^{(\alpha)} G_{\mu}^{($	in a former star, in resultation of the former star of the resultation of the former star	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	5 2 7 2 7 2 7 2 8 2 8 2 2 2 2 2 2 2 2 2 2	$\begin{array}{c} 1 \\ 1 \\ 3 \\ 3 \\ 5 \\ 5 \\ 5 \\ 7 \\ 7 \\ 7 \\ 7 \\ 6 \\ 9 \\ 1 \\ 6 \\ 1 \\ 3 \\ 4 \\ \mathbf{I} \\ \mathbf{I} \\ 1 \\ $

These cutters are made $\frac{1}{32}$ inch larger in diameter and $\frac{1}{64}$ inch greater in thickness than the figures given, to allow for sharpening.

LARGEST SQUARES THAT CAN BE MILLED ON ROUND STOCK

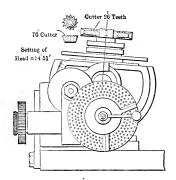
Diam. of Stock	Decimal Equiva- lent	Size of Square	Nearest Fraction	Diam. of Stock	Decimal Equiva- lent	Size of Square	Nearest Fraction
1 8	.125	.088	$\frac{3}{32}$ —	1 9 1 6	1.5625	1.105	164 -
18 2 10 14 5 10 38 2 10 15 15 15 15 15 15 15 15 15 15 15 15 15	.1875	.133	$(\frac{1}{8} + \frac{1}{8})$	I 5/8	1.625	1.149	1 2
14	.250	.177	$\frac{1}{6}\frac{1}{4}$ -	$1\frac{11}{16}$	1.6875	1.193	$1\frac{3}{16} + 1\frac{15}{64} +$
$\frac{5}{16}$.3125	.221	$\frac{7}{32} +$	13	1.750	1.237	$1\frac{1}{6}\frac{5}{4}$ +
<u>3</u>	-375	.265	$\frac{17}{64}$	$1\frac{13}{16}$ $1\frac{7}{8}$	1.8125	1.282	$1\frac{9}{3}$
$\frac{7}{16}$	•4375	.309	5 -	I 7/8	1.875	1.326	$1\frac{21}{64}$ — $1\frac{3}{8}$ —
$\frac{1}{2}$.500	.354	$\frac{23}{64}$ -	$1\frac{1}{1}\frac{5}{6}$	1.9375	1.370	I 3 -
$\frac{9}{1.6}$.5625	.398	$\frac{25}{64}$ +	2	2.000	1.414	$1\frac{1}{3}\frac{3}{2}$ +
. <u>5</u>	.625	.442	$\frac{7}{16} +$	$2\frac{1}{1.6}$	2.0625	1.458	$1\frac{29}{64} +$
$\frac{11}{16}$.6875	.486	$\frac{31}{64}$ +	$2\frac{1}{8}$	2.125	1.502	$1^{\frac{1}{2}} +$
34	.750	.530	$\frac{1}{3}\frac{7}{2}$ -	$ \begin{array}{c} 2\frac{1}{16} \\ 2\frac{1}{8} \\ 2\frac{3}{16} \end{array} $	2.1875	1.547	$1\frac{3}{6}\frac{5}{4}$
$\frac{13}{16}$.8125	.574	$\frac{37}{64}$ -	$2\frac{1}{4}$	2.250	1.591	$1\frac{19}{32}$ — $1\frac{41}{64}$ —
_ 3 _	.875	.619	8 -	$2\frac{5}{16}$	2.3125	1.635	164 -
18	•9375	.663	$\frac{21}{32}$ +	$\begin{array}{c} 2\frac{1}{4} \\ 2\frac{1}{16} \\ 2\frac{3}{16} \\ 2\frac{3}{16} \\ 2\frac{1}{16} \\ $	2.375	1.679	++++
I	1.000	.707	$\frac{45}{64}$ +	$2\frac{7}{16}$	2.4375	1.723	$1\frac{23}{32} +$
$1\frac{1}{16}$	1.0625 .	·755	3 +	$2\frac{1}{2}$	2.500	1.768	1音4 十
$I_{\frac{1}{8}}$	1.125	·795	$\frac{51}{64}$ —	$2\frac{9}{1.6}$	2.5625	1.813	$1\frac{13}{16}$
$1\frac{3}{16}$	1.1875	.840	$\frac{27}{32}$ -	2 8	2.625	1.856	1 6 4 -
$I_{\frac{1}{4}}$	1.250	.884	$\frac{57}{64}$ -	$ \begin{array}{c} 2\frac{11}{16} \\ 2\frac{3}{4} \end{array} $	2.6875	1.900	$1\frac{29}{32}$ —
$1\frac{5}{16}$	1.3125	.928	59 64 +	2 4	2.750	1.944	$1\frac{15}{16} +$
$ \begin{array}{c} 1\frac{3}{16} \\ 1\frac{4}{4} \\ 1\frac{5}{16} \\ 1\frac{3}{8} \\ 1\frac{7}{16} \end{array} $	1.375	.972	$\frac{31}{32}$ +	$ \begin{array}{c c} 2\frac{1}{1}\frac{3}{6} \\ 2\frac{7}{8} \end{array} $	2.8125	1.989	
1 7 8	1.4375	1.016	$1\frac{1}{64} +$	2 7/8	2.875	2.033	$2\frac{1}{32} +$
$\frac{1}{2}$	1.500	1.061	$1\frac{1}{16}$ -	$2\frac{15}{16}$	2.9375	2.077	$2\frac{5}{64}$
Side of Lar	gest Square	=Dia.ofS	tock×.707	3	3.000	2.121	2 1/8 -

Table of Divisions Corresponding to Given Circumferential Distances

This table gives approximate number of divisions and distances apart on circumference, corresponding to a known diameter of work. It is useful in milling-machine work in cutting mills, saws, ratchets, etc.

Dia. of		Distance on Circumference													
Work	35″	16"	1"	3,″ 1 d	1,"	16"	3"	1 ⁷ 6"	12"	9.0	5"	11"	3"	7"	1"
14-5-6-300-7-15-16-10-5-304-1-5-5-14-16-10-4-16-10-4-14-16-10-4-14-16-10-4-14-16-10-4-14-16-10-4-14-16-10-4-16-10-4-16-16-16-16-16-16-16-16-16-16-16-16-16-	327 352 378 402 428 454 478 503 528 554	22 25 31 38 44 50 63 758 100 113 1163 1163 1176 189 201 214 227 239 252 264 277 289	222 3138 4450 5663 6975 8288 94100 114119 1126 1132 1138	6 7 8 10 13 15 17 21 25 29	56 8 9 11 13 16 19 22 25 33 44 47 50 53 57 66 66 67 76	6 8 9 10 13 15 18 20 23 33 35 38 0 43 45 50 55 35 56 1	6 7 8 10 13 15 17 19 21 23 25 27 30 31 4 46 44 46 48 50	6 7 9 11 13 14 16 18 20 22 23 25 27 31 32 34 36 38 40 44 14 44	6 8 10 11 12 14 16 17 19 20 22 24 25 27 28 30 31 33 35 36 38	7 8 10 11 13 14 15 17 18 20 21 22 24 25 27 28 29 31 32 34	6 7 9 10 11 12 14 15 16 18 19 20 21 23 24 25 26 27 28 30	8 9 10 11 13 14 15 16 17 18 19 21 22 23 24 25 26 27	78 9 10 12 13 14 15 16 17 18 19 20 21 22 23 24 25	6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 22	6 7 8 9 9 10 11 13 13 14 15 16 16 17 18

For example: A straddle mill, say, 5 inches in diameter, is to be cut with teeth $\frac{7}{16}$ apart. Without a table of this kind the workman will have to go to the trouble of multiplying the diameter by 3.1416 and then divide by $\frac{7}{16}$ to find the number of teeth to set up for. In the table, under $\frac{7}{16}$ and opposite 5, he can find at once the number of divisions, as 36. Where the table shows an odd number of teeth, one more or less can, of course, be taken if it is important to have even number of teeth.



MILLING SIDE TEETH IN MILLING CUTTERS

THE table gives the angle at which to set the dividing head of a miller when milling the side teeth in milling cutters.

MILLING SIDE TEETH IN MILLING CUTTERS. ANGLE TO SET DIVIDING HEAD

			A	NGLE OF	CUTTER 1	USED		
No. of Teeth	45°	50°	60°	65°	70°	75°	80°	85°
6 77 8 9 10 12 14 16 18 20 22 24 26 28 30 32 34 36 38 40 42	32° 58′ 43° 24′ 54° 44′ 61° 12′ 65° 32′ 72° 55′ 74° 28′ 75° 44′ 76° 32′ 77° 49′ 78° 32′ 79° 51′ 80° 24′ 80° 53′ 81° 20′	32° 57' 45° 15' 52° 26' 66° 02' 66° 10' 72° 13' 73° 44' 77° 04' 77° 04' 77° 58' 80° 23' 80° 23'	43° 36′ 54° 41′ 65° 12′ 70° 32′ 70° 32′ 70° 10′ 77° 52′ 81° 49′ 82° 26′ 81° 49′ 82° 24′ 83° 24	36° 08′ 54° 13′ 66° 36′ 70° 12′ 71° 23′ 71° 52′ 80° 13′ 82° 08′ 83° 33′ 83° 53′ 84° 40′ 85° 32′ 85°	50° 55′ 68° 39′ 72° 13′ 71° 40′ 77° 52′ 78° 20′ 88° 23′ 88° 51′ 88° 51′ 86° 51′ 86° 51′	62° 21' 77° 22' 74° 04' 77° 04' 77° 06' 81° 06' 82° 35' 83° 37' 84° 21' 85° 20' 86° 12' 86° 29', 86° 56' 87° 26' 87° 26'	57° 08' 77° 13' 77° 13' 81° 29' 81° 29' 84° 09' 86° 49' 86° 49' 86° 49' 86° 49' 87° 30' 87° 49' 87° 59' 88° 13' 88° 18' 88° 18' 88° 18' 88° 29'	81° 17' 83° 42' 85° 44' 85° 46' 87° 06' 87° 35' 88° 19' 88° 32' 88° 30' 88° 35' 88° 56' 89° 07' 89° 07' 89° 14'

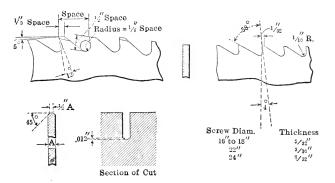
The table shows the angle to the nearest minute, but as an ordinary dividing head is not graduated to read in minutes, the nearest quarter degree is taken.

CUTTING SPEEDS FOR COLD SAW CUTTING-OFF MACHINES

The table on page 204 shows the practice of the Brown & Sharpe Mfg. Co. The semi-high speed still is used for soft steel but high speed is recommended for cutting tool steel. A good lard cutting oil is preferred though it can be dark colored and more impure than for screw machines.

The experience of this company has led to the adoption of the saw tooth shown herewith. This allows a slower speed and coarser feed

than finer teeth, and cuts stock more easily and quickly.



Newton Machine Tool Works recommend a high speed and light feed for steel low in carbon and manganese to keep the chip thin as possible. Up to 35-point carbon use 60 to 65 feet per minute for solid blades. From 35 to 50-point carbon, 55 to 60 feet per minute and less feed. Above 50-point carbon use inserted tooth saws. With inserted tooth saws speeds can be from 50 to 80 feet per minute on 50 to 70-point carbon, but only on heavy, rigid machines. For cutting sprues in steel foundries a solid tooth saw with a speed of 55 feet per minute and a feed of $\frac{1}{4}$ to $\frac{3}{8}$ inch per minute is recommended.

The work should be flooded at all times with any good cutting compound which does not rust. A good mixture is: whale oil, 9 quarts; pure lard oil, 2 gallons; sal soda, 15 pounds; and 40 gallons of water.

The Tindel-Morris Co. give $\frac{3}{4}$ inch per minute as a safe feed for steel bars of 45-point carbon from 2 to 10 inches in diameter. The speed of inserted tooth saws is given as 25 to 30 feet per minute, as this, with a coarse feed, gives good results. This is with $\frac{3}{4}$ inch spacing between teeth. Liberal power is needed — a 36-inch saw requiring from 10 to 15 horsepower.

Table of Cutting Speeds for Cold Saw Cutting-off Machines

	Approx. Min. to Cut Off One Piece	8	IG	12	15.1	10	23	380	33	30	47	۲.	64	74
ol Steel	Revolutions of Saw, per Min.	11.93	11.93	11.03	11.03	10.01	10.01	10.01	10.01	8.68	8.68	7.05	7.95	7.05
High-speed Tool Steel	Feed of Saw, Inches per Min.	0.250	0.250	0.250	0.225	0.210	0.105	0.178	991.0	0.153	0.138	0.126	0.116	0.108
High	Cutting Speed of Saw, Ft. per Min.	50	20	0,5	20	50	0.5	20	20	50	20	20	20	20
	Number of west in Saw	108	108	108	801	801	801	SOI	108	120	120	120	120	120
	Approx. Min. to Cut Off One Piece	3	421	9	73	6	11	13	15	18	21	25	30	35
	Revolutions of Saw per Min.	16.71	16.71	16.71	16.71	14.85	14.85	14.85	14.85	12.15	12.15	11.14	11.14	11.14
Tool Steel	Feed of Saw, Inches per Min.	999.0	0.555	0.500	0.466	0.444	0.400	0.384	0.366	0.333	0.300	0.280	0.250	0.228
	Cutting Speed of Saw, Ft. per Min.	70	70	70	70	70	70	70	20	20	70	70	20	20
	Number of Weeth in Saw	801	801	108	108	108	108	108	108	120	120	120	120	120
	Approx. Min. to Cut Off One Piece	Z = 1	8	4	Ŋ	9	7	- 63 - 63	oi Oi	12	14	17	21	50
Soft Steel	Revolutions of Saw per Min.	11.93	11.93	11.93	11.93	19.01	10.01	19.01	10.01	8,68	89.8	7.95	7.95	7.95
Machinery and Soft Steel	Feed of Saw Inches per Min.	0.88	0.833	0.750	0.70	0.60	0.642	0.588	0.55	0.50	0.404	0.411	0.357	0.307
Mac	Cutting Speed of Saw, Ft. per Min.	50	20	20	20	50	20	20	20	20	20	20	20	20
	Number of Teeth in Saw	92	70	70	70	70	70	20	200	80	8	80	000	00
	Diameter of Saw	91	10	01	01	20	20	10	ΙŞ	22	22	24	24	54
	Diameter of Stock	7	2	ω,	32	4.	42	5	N 7	0	03	7	(Z)	0

TURNING AND BORING

THE accompanying table is a ready means of figuring machine time on turned, bored or faced work.

The ordinary method employed is to ascertain the number of feet in the circumference of the piece by multiplying the diameter in inches by 3.1416 and dividing by 12. The next step consists of dividing the length by the feed used, which gives the entire number of revolutions the piece must make.

CONSTANTS FOR CUTTING TIME, IN MINUTES

FEED IN INCHES		CUTTING SPEED										
Two Cuts at	15 Feet	18 Feet	20 Feet	25 Feet	30 Feet	35 Feet	40 Feet					
I-64 I-32 I-32 I-16 I-16 I-8 I-8 I-4	1.676 0.8378 0.4189 0.2094	1.396 0.6981 0.3491 0.1745	1.257 0.6283 0.3142 0.1571	1.005 0.5027 0.2513 0.1257	0.8378 0.4189 0.2094 0.1047	0.7181 0.3590 0.1795 0.0898	0.6281 0.3141 0.1570 0.0785					
One Cut at 1-64 1-32 1-16 1-8 1-4	1.117 0.5585 0.2792 0.1396 0.0698	0.9308 0.4654 0.2327 0.1164 0.0582	0.8378 0.4189 0.2094 0.1047 0.0524	0.6702 0.3351 0.1676 0.0838 0.0419	0.5585 0.2793 0.1396 0.0698 0.0349	0.4787 0.2394 0.1197 0.0598 0.0299	0.4189 0.2094 0.1047 0.0524 0.0262					
Two Cuts at	45 Feet	50 Feet	60 Feet	70 Feet	80 Feet	90 Feet	100 Feet					
I-64 I-32 I-32 I-16 I-16 I-8 I-8 I-4 One Cut at	0.5581 0.2793 0.1396 0.0698	0.5027 0.2513 0.1257 0.0683	0.4189 0.2094 0.1047 0.0524	0.3590 0.1795 0.0898 0.0449	0.3142 0.1571 0.0785 0.0393	0.2793 0.1396 0.0698 0.0349	0.2513 0.1257 0.0628 0.0314					
1-64 1-32 1-16 1-8 1-4	0.3723 0.1862 0.0931 0.0465 0.0233	0.3351 0.1676 0.0838 0.0419 0.0209	0.2793 0.1396 0.0698 0.0349 0.0175	0.2394 0.1197 0.0598 0.0299 0.0150	0.2094 0.1047 0.0524 0.0262 0.0131	0.1852 0.0931 0.0465 0.0233 0.0116	0.1676 0.0838 0.0419 0.0209 0.0105					

Having the circumference in feet and the number of revolutions, it is necessary to multiply them to find the entire number of feet traveled. Dividing this result by the speed of cut in feet per minute will give the actual cutting time in minutes.

The figuring is not complicated in any way, but it has the disadvantage of taking too much time. It can be resolved into the follow-

ing formula:

$\frac{\textit{Diameter} \times 3.1416 \times \textit{length in inches}}{12 \times \textit{speed} \times \textit{feed in inches}} = \textit{time in minutes}.$

The known factors in the case can be resolved into a constant which is directly dependent on the feed and speed; hence, a table covering a wide range of speeds and feeds is necessary for their proper use.

One part of the table gives constants for two cuts at $\sqrt[3]{4}$ -inch and $\sqrt[3]{2}$ -inch feed, up to $\sqrt[3]{8}$ -inch and $\sqrt[4]{4}$ -inch feed, at any speed from 15 feet to 100 feet per minute. The rest of the table gives constants for one cut at $\sqrt[3]{4}$ -inch up to and including $\sqrt[4]{4}$ -inch feed, also at any speed from 15 feet to 100 feet per minute.

A typical computation is as follows:

A piece 4 inches in diameter, 10 inches long, is turned with two cuts at $\frac{1}{16}$ -inch and $\frac{1}{32}$ -inch feed, each with a cutting speed of 20 feet per minute. Diameter \times length in inches \times constant = time in min-

utes. $4 \times 10 \times 0.6283 = 26$ minutes.

If, for the purpose of accuracy it is thought advisable, in connection with these two cuts, to run another cut over, the constant 0.6283 is added to a constant for the third feed used. If this feed is $\frac{1}{64}$ -inch, then we have 0.6283 + 0.8378 = 1.4661, the constant for three cuts, one of $\frac{1}{16}$ -inch, one of $\frac{1}{42}$ -inch, and one of $\frac{1}{64}$ -inch feed.

TABLE HAS WIDE APPLICATION

There is hardly a combination of feeds and speeds that it is not possible to secure by inspection from the table. By interpolation an added number can be secured.

The table can be adapted, on account of its wide range, to the known individual performance of any lathe or boring mill in the shop. No slide rule or any special operations are necessary to secure the desired results, merely a knowledge of multiplication.

For flange facing it is possible to use the table with the same ease

as for boring and turning by figuring on the main diameter.

ROTARY CUTTING SPEED

An easy method of calculating the cutting speed of a lathe tool or milling cutter is to divide the number of revolutions by 4 and multiply by the diameter in *inches*. This gives the cutting speed in *feet* per minute. Dividing 3.1416 by 12 gives .262 and .25 is very close after allowing for belt slip.

Let D = diam. of work, cutter or boring bar.

N = revolutions per minute. C = cutting speed in feet per minute.

$$C = D \times \frac{N}{4}$$
 $N = \frac{4 \times C}{D}$ $D = \frac{4 \times C}{N}$

LATHE TOOL TESTS

In testing steels for lathe and similar tools it has become customary to use standard material, speeds, feeds and depth of cut. In some cases tools are run until they break down, in others they are

passed if they stand up for a specified time. A 20-point carbon steel is often selected and cuts $\frac{1}{4}$ inch deep with $\frac{1}{16}$ to $\frac{1}{8}$ inch feed at cut-

ting speeds of from 60 to 90 feet per minute.

The U. S. Navy Department specifies that a $\frac{7}{8}$ -inch lathe tool shall stand up for 20 minutes with a $\frac{3}{16}$ -inch cut and $\frac{1}{16}$ -inch feed at 60 feet per minute without regrinding; the material must be at least 80,000-lb. tensile, and 50,000-lb. elastic limit with a 25 per cent. elongation in 2 inches. The steel is annealed before the test.

CUTTING LUBRICANTS

Cast iron is usually worked dry, but when hard cast-iron gears are to be cut, as with three cutters, the first cut through will work better with strong soda water. It makes an objectionable mess, but the work will be done faster and the cutters keep sharper longer than with the dry process of cutting.

Brass and babbitt are usually cut dry, but to hand-ream brass and babbitt is sometimes difficult if the reamer is a little dull. Kerosene and turpentine are used with good results. Cast iron can be handreamed easily with tallow and graphite, mixed, and the hole will be kept just the size of the reamer. Copper can be worked well with lard-oil and turpentine mixed.

In boring babbitt bushings and rod boxes in a lathe or boring mill, it is very difficult to work the material dry as the chips have a great tendency to roll around the tool and into a hard ball, tearing the metal and making a rough ragged hole. In this case kerosene and

lard-oil mixed will work well.

Cheap oil is sometimes used as a lubricant for cutting, but soap water or soda water is better for iron and steel shafting and with a sharp tool and light finish cut the work will be smooth enough to

polish without filing.

Rawhide is a very peculiar substance to work, and to drill it with a twist drill is a tedious job, as the flutes will clog and stick if run dry. A cake of soap held against the drill will prevent all trouble and sticking of drills. It is bad practice to use oil on rawhide as it injures the fiber and loosens the glue. Drills should be run at very high speed in rawhide to work well.

Turpentine is good in some cases where fitting is done, such as scraping lay-out plates, or face plates. Oil will form a coating so that marks cannot be seen plainly, but turpentine will prove beneficial on this kind of work if used freely. The marks can be seen plainly, and the work is a great deal easier to scrape than with an oil surface, as the oil glazes over the surface and makes it hard to

start a tool.

GRINDING AND LAPPING

GRINDING WHEELS AND GRINDING

The Commercial Abrasives

EMERY, corundum, carborundum, and alundum are the ordinary commercial abrasive materials. They vary in hardness, though it does not follow that the hardest grit is the best for cutting purposes; the shape and form of fracture of the particles must also be taken into consideration. We may imagine a wheel made up from diamonds, the hardest substance in nature, and whose individual kernels were of spherical form; it is quite obvious that it would be of little service as a cutting agent; on the other hand, if these kernels were crystalline or conchoidal in form it would probably be the ideal grinding wheel.

Emery is a form of corundum found with a variable percentage of impurity; it is of a tough consistency and breaks with a conchoidal

fracture.

Corundum is an oxide of aluminum of a somewhat variable purity according to the neighborhood in which it is mined; its fracture is conchoidal and generally crystalline.

Carborundum is a silicide of carbon and is a product of the electric

furnace; it breaks with a sharp crystalline fracture.

Alundum is an artificial product, being a fused oxide of aluminum. It is of uniform quality with about 98 per cent. of purity. It breaks with a sharp, conchoidal crystalline fracture and has all the toughness of emery.

Grit and Bond

A GRINDING wheel is made up of the "grit" or cutting material, and the bond. The cutting efficiency of a wheel depends largely on the grit; the grade of hardness depends principally on the bonding material used. The efficiency in grinding a given metal is dependent largely upon the "temper," or resistance to fracture, and, as noted above, upon the character of fracture of the grit or cutting grains of the wheel.

The function of the bond is not only to hold the cutting particles of the wheel together and to give the wheel the proper factor of safety at the speed it is to be run, but it must also be possible to vary its tensile strength to fit the work it is called upon to do. We often hear the operator say that the wheel is too hard or too soft. He means that the bond retains the cutting teeth so long that they become dulled, and this wheel is inefficient; or, in the case of a soft wheel, the bond has not been strong enough to hold the cutting teeth and they are pulled out of the wheel before they have done the work expected.

The bond to be used for a given operation depends on the wheel and work speeds, area of wheel in contact with the work, vibration in wheel spindle or work, shape and weight of work, and many other like variables.

Wheels are bonded by what are known as the vitrified, silicate, elastic and rubber processes. No one bond makes a superior wheel

for all purposes; each one has its field.

The vitrified bond is made of fused clays, is unchanged by heat or cold, and can be made in a greater range of hardness than any other bond. It does not completely fill the voids between the grains, and, therefore, a wheel bonded in this way having more clearance than any other, is adaptable for all kinds of grinding except where the wheel is not thick enough to withstand side pressure. This bond has no elasticity.

The silicate bond is composed of clays fluxed by silicate of soda at low temperatures. It is not as stable as the vitrified bond as regards dampness, gives less clearance between grains, and has a range of hardness below that of the vitrified in the harder grades. This bond has no elasticity and will not make a safe wheel of extreme thinness.

The elastic bond is composed of shellac and other gums. It completely fills the voids of the wheel, has a limited range of grades, has a high tensile strength and elasticity, and can be used for the making of very thin wheels. The rubber or vulcanite bond has the general characteristics of the elastic, but its grades of hardness cannot be varied to the same extent and its uses are limited.

Grain and Grade

Grinding wheels are made in various combinations of coarseness and hardness to meet the variety of conditions under which they are used. The cutting material is crushed and graded from coarse to fine in many sizes designated by number. Thus the sizes of grain used in the Norton wheels are numbered 10, 12, 14, 16, 20, 24, 30, 36, 46, 50, 60, 70, 80, 90, 100, 120, 150, 180, 200. Finer grades known as flour are also used, sometimes these being designated as F, FF, FFF, etc. By No. 20 grain is meant a size that will pass through a grading sieve having 20 meshes to the linear inch.

The term "grade" refers to the degree of hardness of the wheel or the resistance of the cutting particles under grinding pressure. A wheel from which the cutting particles are easily broken, causing it to wear rapidly, is called soft, while one which retains its particles

longer is called hard.

Minimum Thickness of Wheels

Tables 1, 2, and 3 by the Norton Company, show minimum thickness of wheels made by the different processes. A wheel of fine grit can be made thinner than a wheel of coarse grit, and have the same factor of safety. The minimum thickness depends upon both the diameter and the coarseness of the grit used.

For example, a 24-inch vitrified wheel of No. 10 or No. 12 grain should not be made thinner than 2 inches, while if a grain No. 36 or

finer is used it is considered a safe wheel at 1 inch thick.

Table 1. — Minimum Thickness of Elastic Wheels

			Grain		
	14 and 16	20 and 24	30 and 36	46 and 60	70 and finer
Diameter in Inches		Minimum Th	ickness of Wl	neels in Inche	S
28 to 30 24 to 26 20 to 22 16 to 18 14 to 15 10 to 12 7 to 9 5 to 6 3 to 4 1 to 2	3458 1238 145 8 6 8 1 5 1 5 1 5 1 6 8 1 6	3) 4 5) (8) 1-(2 1 2) (6) 1-(4 5) (7) (7) (8) 1-	3 4 -2 2 6 4 3 6 5 6 1 1	34 4 2 2 3 8 1 4 4 3 6 1 1 6 5 1 1 6 5 1 7 1 6 5 1 7 2 7 3 2 2	3.4 1.7 2.3 8.1 1.4 4.3 1.6 1.7 1.6 1.7 1.7 1.7 1.7 1.7 1.7 1.7 1.7 1.7 1.7

Table 2. — Minimum Thickness of Vitrified Wheels

,		Grain										
	10 and 12	14 and 16	20 and 24	30	36	46	50 to 120	150 and fine				
Diameter in Inches		Minimum Thickness of Wheel in Inches										
32 to 36 24 to 30 20 to 22 16 to 18 14 to 15 10 to 12 7 to 9 5 to 6 3 to 4 1 to 2 Smaller than I In.	$1\frac{1}{2}$ $1\frac{1}{2}$	2 1 2 1 2 1 2 1 2 1 2 1 4 1 1 4 1 1 1 1	2 I I I স্বাধ্য সংগ্ৰহণৰ স্থান্ত সংগ্ৰহণৰ সংগ্ৰহণৰ স্থান্ত সংগ্ৰহণৰ স্থান স্থান্ত সংগ্ৰহণৰ স্থান স্থান্ত সংগ্ৰহণৰ স্থান স্	2 I 1414 I 3458388388388388388	2 I I 3 4 3 4 1 6 2 5 6 1 1 4 1 4 1 4	2 I 3 4 3 4 1 2 3 8 1 4 1 4 8 1 6 1 8	2 I S 41-533 63 63 61-141-161-161-161-161-161-161-161-161-1	2 I 344-12300-144-144-56-168 I 16				

TABLE 3. - MINIMUM THICKNESS OF SILICATE WHEELS

					G	rain						
	10, 12, 14, and 16	20 a	nd 24		30		36	40 a	nd 50	60 to 120	and fine	
		Wire Web										
	With With out	With	With- out	With	With- out	With	With- out	With	With- out	Wit	hout	
Diameter in Inches		N	Iinimu	ım Th	ickness	of W	neel in	Inche	3			
44 to 48	3	3	2	3	2	3	2	3	2	2	2	
38 to 42	2 1/2	$2\frac{1}{2}$	2	$2\frac{1}{2}$	2	2 1/2	2	$2\frac{1}{2}$	2	2	2	
32 to 36	2 1/2	$2\frac{1}{2}$	2	2 1/2	I 1/2	2 1/2	1 1/2	$2\frac{1}{2}$	I 1/2	$1\frac{1}{2}$	$I_{\frac{1}{2}}$	
28 to 30	2	2	2	2	2	2	1	2	1	I	I	
24 to 26	2	$I^{\frac{1}{2}}$	I 1/2	2	I	2	1	2	I	I	I	
20 to 22	I 3/4	$I^{\frac{1}{2}}$	$I^{\frac{1}{2}}$	I 1/4	I	I 1/4	1	I 4	I	1	I	
16 to 18	I 1/2	I	I	1	3 4	1	34	I	3 4	34	3	
14 to 15	I ½	1	I	I	3 4	1	3 4	I	3 4 3 6 3 8	3 4 3 8 3 8	34	
10 to 12	I ½	3 4 3 4	34	3 4 3 4	1/2	3 4	1/2	3 4 3	3 8	3 8	38	
7 to 9		3	3 4	34	1/2	3 4	1/2	34	38		3 8	
5 to 6					3/4 1/2 1/2 1/2 1/2 1/2		1 4		14	1 4	3 4 3 8 3 8 1 4 1 8 1 8	
3 to 4					1/2		1 4		1/4	1 2	18	
I to 2			ĺ	1	1 2		1 1		18	1 8	18	

Grading of Wheels

OF the many firms engaged in the manufacture of grinding wheels there are probably no two which have a similar method of grading or designating the hardness of their wheeels. The Norton Company, which is probably the oldest in the field, uses the letter method, which may be said to be the simplest. That is, they take M for their medium-hard wheel and the letters before M denote in regular alphabetical progression the progressively softer wheels. Moreover they use a + mark for denoting wheels which vary in temper from the standards. Thus a wheel may be harder than the standard K, and still be not so hard as the standard L; in this case it is known as K +. The Carborundum Company adopts a somewhat similar method of grading, the difference being that although M denotes its mediumhard wheel the letters before M denote the progressively harder grades. Various other American companies use the letter method of grading to some extent, but all have individual ideas as to what degree of hardness should constitute an M or medium-grade wheel. Then there are firms both in America and on the continent of Europe which discard the letter method of grading or else use it in conjunction with numbers or fractions of numbers such as 2H, 11M and so on.

The selection of suitable wheels for machine grinding may be said to be governed by the following points, namely, the texture of the material to be ground, the arc of wheel contact with work and the quality of finish required. The first and last of these points can for convenience' sake be taken in conjunction. The quality of surface finish is dependent on the condition of the wheel face and depth of cut rather than on the fineness of the grit in the wheel. A wheel of so fine a grit as 100 will give an indifferent finish if it is not turned true and smooth.

It may be assumed that for all general purposes the aim in view is to procure a wheel which will fulfil two conditions, that is, that it shall first remove stock rapidly and at the same time give a decent finish. Wheels made from a combination of grit of different sizes are the best for this purpose, as may be seen from the following explanation. Coarse wheels of an even number of grit will remove stock faster than will fine wheels of an even number, because their depth of cut or penetration is greater. They, however, fail in giving a high surface finish except in grinding very hard material, because they are not compact enough.

The Combination Grit Wheel

WITH the combination wheel the conditions are different and it seems better at removing stock than does the coarse, even grit wheel. It may be safe to assume from this that something of a grindstone action takes place, that is, that the finer particles of grit become detached from the bond and both roll and cut in their imprisoned condition between the larger particles. For finishing purposes this wheel has all the compactness and smooth face of a wheel which was made solely from its finest number of grit; and for roughing, it enables a depth of cut to be got which is within the capacity of its largest kernels.

With regard to the texture or hardness of material ground it may be taken as a general rule that the harder the material is, the softer the bond of wheel should be, and that cast iron and hardened steel bear some relation to each other as far as grinding wheels are concerned, for the same wheel is usually suitable for both materials.

Too large an assortment of wheels is likely to lead to confusion and we may take the Norton plain cylindrical grinding machine as being a case in point of a limited assortment of wheels; at the same time it will be a starting point to illustrate choice of wheels under various grinding conditions. In this machine four different grade wheels, all of 24 combination grit, are found sufficient for all classes of material that it is ordinarily required to grind. These include high- and low-carbon steels, cast iron, chilled iron, and bronze or composition metals. These wheels are graded J, K, L, and M.

Hard Wheels

ONE of the greatest advantages accruing from grinding is that it ignores the non-homogeneity of material and that it machines work with the lightest known method of tool pressure, thus avoiding all deflections and distortions of material which are a natural result of

the more severe machining processes. Yet these objects are too often defeated by the desire for hard and long-lived wheels. A wheel that is too hard or whose bond will not crumble sufficiently under the pressure of cut will displace the work and give rise to many unforeseen troubles. It is also a prolific cause of vibration which is antagonistic to good and accurate work. The advantage claimed for it, that it gives a better surface finish, is a deceptive one, for it mostly obtains this finish at the expense of accuracy. Quality of finish, that is, accurate finish, is merely a question of arranging of work speed, condition of wheel face and depth of cut. In the machine mentioned the suitability of wheels to materials and conditions is found to be as follows, the wheels being in each case of a combination of alundum grit:

For hard chilled iron and large diameters of cast iron and hardened steel

For all grades of steel which are not hardened and for bronze 24 L
For very low carbon machine steels 24 M
The table given may appelling appearably be what would be about

The table given may, speaking generally, be what would be chosen in the way of wheels for the materials given, and in actual practice they soon give evidence as to whether they are suitable. It may be gathered from the table that diameter of work is a factor in the choice of a wheel. This refers to area of wheel contact and is governed by what is shown in the table when broad differences of diameter occur; for instance, it might be necessary to use the K wheel for a large diameter of high carbon steel if the L wheel was evidently too hard.

Speed and Efficient Cutting

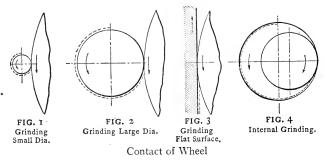
The efficient cutting of a wheel depends very much on the speed of the work, and an absence of knowledge in this respect may often lead to a suitable wheel's rejection. Revolving the wheel at the speed recommended by the maker is the first necessity, and if it is found unsuitable after experimenting with various speeds it should be changed for a softer or harder one as the conditions indicate. Starting from the point that a wheel is desired that shall remove the maximum amount of stock with the minimum amount of wear on the wheel, the indications and method of procedure may be as follows; only it must be understood that this refers to cases where an ample supply of water is being delivered at the grinding point.

If, after trying all reasonable work speeds, a wheel should burn the work, or refuse to cut without excessive pressure, or persistently glaze the surface of the work, it is too hard for that particular work and material and may be safely rejected. If, after trying all reasonably reduced work speeds, a wheel should lose its size quickly and show all signs of rapid wear, it is too soft for that particular work and material and may be rejected. These indications refer to all ordinary cases and it may be gathered that the most economical wheel is that which acts in such a manner as to be a medium between the two cases. There is still another point to bear in mind with regard to the size of the grit in the wheel, but which refers more especially

to very hard materials such as chilled iron. Either a coarse or combination wheel may go on cutting efficiently in roughing cuts because pressure is exerted, but may begin to glaze when this pressure is much relieved as in finishing cuts. A careful microscopic scrutiny of a wheel that displays this tendency would seem to lead to the following assumption:

When a Wheel is Sharp

THE wheel face when newly trued with the diamond tool, which is necessary to obtain an accurate finish, shows a promiscuous arrangement of particles, some of which present points and others present a broader face with a rough and granular surface. When the wheel is presented to the hard surface of the work the high points of this granular face and the sharp contour of the kernels will go on cutting until they are dulled and worn down, after which their face



area is too great to enter the surface without undue pressure. When the wheel has reached this condition the microscope shows these broader-faced kernels polished to a metallic luster, which bears out the explanation tendered and also makes the remedy quite apparent. This is to use a wheel of very fine grit for finishing purposes in these cases or else keep the coarser wheel in condition by repeated dressings with the diamond tool.

Wheel Contact

REFERENCE to Figs. 1 to 4 will show what actual practice requires in the choice of a wheel so far as the question of wheel contact is concerned. A wheel is shown in contact with four different varieties of work, all of which we will suppose to be of the same material, the depth of cut, much exaggerated, being the same in each case. In the first case it is a shaft of small diameter, and the wheel contact being the smallest the harder grade of wheel would be suitable, comparatively speaking. Assuming that this wheel was found to be suitable it would probably require a softer wheel for the next case, which is a shaft of larger diameter, and the wheel contact proportionately greater. To continue the comparison still further, the third

case shows the wheel engaged in grinding a flat surface, and the fourth is a wheel grinding internally. In each case practice demands that the wheel shall be progressively softer in bond or grade and is some proof of a consistency in the action of grinding wheels.

The Contact Area of a Wheel

THE most probable explanation of this may be that as the contact area increases more work is required from each individual kernel of grit and it the sooner becomes dulled; this requires that the bond must be more friable both to allow it to escape easily and to minimize the pressure required to make the wheel cut as the cutting area becomes greater. Following on this reasoning we are able to choose a list of wheels which would be suitable for almost all purposes, and which would be as follows if of Norton grade:

For plain cylindrical grinding $J \ K \ L \ M$ For grinding plane surfaces $H \ I \ J \ K$ For internal grinding $F \ H \ I \ J$

This collection of wheels would be suitable for almost any type of grinding machines, though when the wheels are exceptionally narrow a grade or one-half grade higher might be possible; it would, of course, be a matter for a little trial and experiment. The wheels for external cylindrical work may preferably be combination wheels, but for plane surface and internal work they are better made of single grit, about 36 or 46. The great contact area of wheel in these two classes of work is liable to generate much heat so that an open and porous wheel is preferable.

Wheel Pressure and Wear

As the wheel is a disk built up from a numerous assortment of minute cutting tools which are held in position by a more or less friable bond, in using it we must bring it to bear on the work with a pressure that shall not be so great as to tear these minute tools from their setting until their cutting efficiency is exhausted, for if we do so we are wasting the wheel. To gage the exact amount of the pressure required is a matter of judgment and experience, though where automatic feeds are provided on a machine the right amount of pressure or feed is soon determined. It will also be readily understood that a regular automatic feed is more reliable for the purpose than a possibly erratic hand one. The automatic feed may be set to give a certain depth of cut at each pass of the wheel, and its amount of wear noted; if this wear be found excessive the depth of cut may be reduced. It must not be here forgotten that work speed also enters into this consideration and that a high work speed will tend to wear the wheel excessively; inversely a reduced work speed will reduce the amount of wear. Having these points in mind the right combination of depth of cut and work speed is soon arrived at, and an approximate judgment attained for the future.

Grinding Allowances

THE amount of stock left for removal by the grinding wheel and the method of preparing the work have both much bearing on the economic use of grinding wheels, and heavy and unnoticed losses often occur through want of a few precautionary measures. The necessary amount of stock to leave on a piece of work as a grinding allowance depends firstly on the type of machine employed, the class of labor engaged in preparing it, and whether it has to be hardened or otherwise.

In powerful machines, which will remove stock rapidly, the grinding allowance may be anything up to $\frac{1}{32}$ inch. There are many cases of an especial character when the grinding allowance stated may be exceeded to advantage so long as discretion is used. Straight shafts may often be ground direct from the black bar of raw material $\frac{1}{16}$ inch above finished size, or when shafts of this character must have large reduction on the ends they can be roughly reduced in the turret lathe while in their black state and finished outright more economically in the grinding machine. Very hard qualities of steels or chilled rolls are other cases where it is often more economical to use the grinding machine without any previous machining process, and though there may be sometimes an alarming waste of abrasive material its cost is as nothing compared with other savings that are made.

Grinding allowances for hardened work are usually larger than for soft work, to allow for possible distortion; so that individual experience alone can determine the amount to be left. It is sufficient to say that the allowances on case-hardened or carbonized work should not be excessive; otherwise the hardened surface may be ground away.

Grinding Hardened Work

As far as the actual grinding of hardened work goes, it is indispensable that the whole portion of a piece that is to be ground should be roughed over previous to the final finishing; if it is at all possible to allow some little time to elapse between the two operations so much the better, more especially if it has bent in hardening and been afterward straightened; this will allow of the development of any strain that may be present. Both for special and standard work in a factory a table of grinding allowances can be compiled as a result of experience and posted in a conspicuous position. If this be done and trouble taken to see that it is adhered to, it will save much trouble and be a means of avoiding much unnecessary expense.

It is necessary to slightly undercut the corners of shoulders so as to preserve the corner of the grinder's wheel intact. A piece of work should never be prepared in such a manner as to form a radius on the corner of the wheel, for to get the wheel face flat again means much waste of wheel and wear of diamond. Where fillets or radii are necessary they are better got out with a tool, for even if they are to be ground they must be turned good to allow the wheel to conform to their shape. The only excusable reason for grinding a round corner is when the work is hardened or in some special case where the expense incurred is warranted.

GRINDING ALLOWANCES FOR VARIOUS LENGTHS AND DIAMETERS

Table 4 shows the practice of the Landis Tool Company, in reference to grinding allowances. This table covers work up to 12 inches diameter, and lengths to 48 inches.

Table 4. — Allowances for Grinding (Landis Tool Co.)

3"	6"	9"	12"	15"	18"	24"	30"	36"	42"	48"
.010	.010	.010	.010	.015	.015	.015	.020	.020	.020	.020
.010	.010	.010	.010	.015	015	.015	.020	.020	.020	.020
.010	.010	.010	.015	.015	.015	.015	.020	.020	.020	.020
.010	.010	.015	.015	.015	.015	.015	.020	.020	.020	.020
.010	.015	.015	.015	.015	.015	.020	.020	.020	.020	.020
.015	.015	.015	.015	.015	.020	.020	.020	.020	.020	.025
.015	.015	.015	.015	.020	.020	.020	.020	.020	.025	.025
.015	.015	.015	.020	.020	.020	.020	.020	.025	.025	.025
.015	.015	.020	.020	.020	.020	.020	.025	.025	.025	.025
.015	.020	.020	.020	.020	.020	.025	.025	.025	.025	.025
.020	.020	.020	.020	.020	.025	.025	.025	.025	.025	.030
.020	.020	.020	.020	.025	.025	.025	.025	.025	.030	.030
.020	.020	.020	.025	.025	.025	.025	.025	.030	.030	.030
.020	.020	.025	.025	.025	.025	.025	.030	.030	.030	.030
.020	.025	.025	.025	.025	.025	.030	.030	.030	.030	.030
.025	.025	.025	.025	.025	.030	.030	.030	.030	.030	.030 -
.025	.025	.025	.025	.030	.030	.030	.030	.030	.030	.030
.025	.025	.025	.030	.030	.030	.030	.030	.030	.030	.030
.025	.025	.030	.030	.030	.030	.030	.030	.030	.030	.030
.030	.030	.030	.030	.030	.030	.030	.030	.030	.030	.030
	.010 .010 .010 .015 .015 .015 .020 .020 .020 .025	.010 .010 .010 .010 .010 .015 .015 .015	.010 .010 .010 .010 .010 .010 .010 .010	.010 .010 .010 .010 .010 .010 .010 .010 .010	.010 .010 .010 .010 .015 .010 .010 .010 .015 .010 .010 .010 .015 .015 .010 .010 .015 .015 .010 .010 .015 .015 .010 .015 .015 .015 .015 .015 .015 .015 .015 .020 .015 .015 .015 .020 .020 .015 .015 .020 .020 .020 .015 .020 .020 .020 .020 .020 .020 .020 .020	.010 .010 .010 .010 .015 .015 .010 .010 .010 .015 .015 .010 .010 .010 .015 .015 .010 .010 .010 .015 .015 .010 .010 .015 .015 .015 .010 .015 .015 .015 .015 .015 .015 .015 .015 .015 .020 .020 .015 .015 .015 .015 .020 .020 .015 .015 .020 .020 .020 .015 .020 .020 .020 .020 .020 .020 .020 .020	.010 .010 .010 .010 .015 .015 .015 .015	.010 .010 .010 .010 .015 .015 .015 .020 .020 .010 .010 .010 .015 .015 .015 .020 .010 .010 .010 .015 .015 .015 .020 .010 .010 .010 .015 .015 .015 .020 .020 .010 .010 .015 .015 .015 .015 .020 .020 .010 .015 .015 .015 .015 .015 .020 .020 .020 .025 .015 .015 .015 .015 .015 .015 .015 .01	.010 .010 .010 .010 .015 .015 .015 .020 .020 .020 .010 .010 .010 .015 .015 .015 .020 .020 .020 .010 .010 .015 .015 .015 .015 .020 .020 .020 .010 .010 .015 .015 .015 .015 .020 .020 .020 .010 .010 .015 .015 .015 .015 .015 .020 .020 .020 .010 .015 .015 .015 .015 .015 .020 .020 .020 .015 .015 .015 .015 .015 .020 .020 .020 .021 .015 .015 .015 .015 .015 .020 .020 .020 .021 .015 .015 .015 .015 .015 .015 .020 .020 .020 .020 .021 .015 .015 .015 .015 .015 .020 .020 .020 .020 .021 .015 .015 .015 .015 .015 .020 .020 .020 .020 .020 .020 .020 .02	.010 .010 .010 .010 .015 .015 .015 .015

OTHER GRINDING ALLOWANCES

TABLE 5 gives the allowances of the Brown & Sharpe Mfg. Co. in rough turning work for the grinding department. Limit gages of the form shown are used, the dimensions in the table covering work up to 2 inches diameter.



Table 5.—Limit Gage Sizes for Lathe Work which is to be Finished by Grinding

(Brown & Sharpe Mfg. Co.)

Size	Not go on	Go on	Size	Not go on	Go on	Size	Not go on	Go on		
	Inches			Inches		Inches				
387781298 1581483843878	0.383 0.4455 0.508 0.5705 0.633 0.6955 0.758 0.8205 0.883	0.387 0.4495 0.512 0.5745 0.637 0.6995 0.762 0.8245 0.887	$\begin{array}{c} 15 \\ 16 \\ I \\ $	0.9455 1.008 1.0705 1.133 1.1955 1.258 1.3205 1.383 1.4455	0.9495 1.012 1.0745 1.137 1.1995 1.262 1.3245 1.387 1.4495	$\begin{array}{c} \mathbf{I} & \frac{1}{2} \\ \mathbf{I} & \frac{1}{16} \\ \mathbf{I} & \frac{5}{8} \\ \mathbf{I} & \frac{1}{16} \\ \mathbf{I} & \frac{3}{4} \\ \mathbf{I} & \frac{1}{16} \\ \mathbf{I} & \frac{1}{16} \\ \mathbf{I} & \frac{1}{16} \\ \mathbf{I} & \frac{1}{2} \\ \mathbf{I} & \frac{1}$	1.508 1.5705 1.633 1.6955 1.758 1.8205 1.883 1.9455 2.008	1.512 1.5745 1.637 1.6995 1.762 1.8245 1.8875 1.949 2.012		

Use of Water

WATER should be applied at the right spot. This spot must be right at the *grinding point*, whether it be internal, external, or plane-surface work, and must be delivered with sufficient force to keep the wheel face clean. If this is not done there is a kind of mud accumulated at the grinding point, which causes glazing. Water is, or should be, used in grinding process not as a means of quenching heat but rather to prevent its creation and radiation, and so the actual grinding point is the best place to apply it.

It is a necessary means of keeping the work at an equable temperature so as to obviate distortion and to make the matter of taking dimensions an actuality rather than a guessing matter. This

applies equally to all kinds of grinding.

The Use of Diamonds

HERE it is perhaps well to give the question of diamonds some little consideration as they are sometimes a very expensive item. A diamond is a very essential part of a grinding machine's equipment, for in its absence a good and highly finished grade of work is an impossibility. It is perhaps unnecessary to state that they should be the hardest rough stones procurable, and that the larger they are the cheaper they are in the end. With regard to their size: This is a

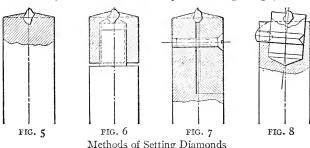
known proportionate element in their price per carat, but a large stone allows of a more secure hold in its setting and so the danger of losing it is reduced. As a further precaution against this danger the diamond tool should always be held by mechanical means when using it except in cases which are unavoidable; this may be in cases where profile shapes have to be turned on the wheel face. An attempt to turn by hand a perfectly flat face on a wheel, which is necessary for finishing, must of a necessity end in failure.

As a means of preservation of the diamond a full stream of water should be run on it when in use and many light chips are preferable to a few heavy ones. The main thing is to watch that it does not get unduly heated, for this is disastrous to it. Where large quantities of material have to be removed from a wheel the ordinary wheel dresser may be employed to reduce the bulk of the stock, and the

diamond only used for finishing to shape.

Setting the Diamonds

DIAMONDS may be obtained ready fixed in suitable holders or the rough stones may be bought and set by any competent toolmaker. The illustrations show various methods by which they may be held securely and require but little explanation. First, Fig. 5 is the method most commonly used, the diamond being either peened or brazed in position. One disadvantage of this method is that the diamond is apt to break with a chance blow of the peening chisel, or the heat from brazing will sometimes cause fractures; neither is it so easily reset when its point becomes dulled as are the other methods shown. Fig. 6 requires no explanation except that it is advisable to pack the diamond with shredded asbestos fiber to act as a cushion; this method allows of quick resetting. Fig. 7 consists



of a small steel cap tapped out to fit the stock as shown. Enough shredded asbestos fiber is inserted between the diamond and stock to hold it firmly in position. This method also allows of quick and safe resetting. The fourth method, Fig. 8, is covered by patent rights and its advantage can be seen at a glance; as the diamond wears, the small peg containing it can be revolved in the stock to present a new cutting edge and be so clamped in position.

Speed Tables, Rules for Surface Speeds, etc.

THE table below gives the number of revolutions per minute at which grinding wheels of diameters ranging from 1 to 60 inches must be operated to secure peripheral velocities of 4000, 5000, 5500 and 6000 feet per minute. Ordinarily a speed of 5000 feet per minute is employed, though sometimes the speed is somewhat lower or higher for certain cases.

GRINDING WHEEL SPEEDS

Diameter Wheel	Rev. per Minute for Surface Speed of 4000 ft.	Rev. per Minute for Surface Speed of 5000 it.	Rev. per Minute for Surface Speed of 5500 ft.	Rev. per Minute for Surface Speed of 6000 ft.
I Inch 2 Inches 3 Inches	15,279 7,639 5,093	19,099 9,549 6,366	21,000 10,500 7,350	22,918 11,459 7,639
4 Inches 5 Inches 6 Inches 7 Inches	3,820 3,056 2,546 2,183	4,775 3,820 3,183 2,728	5,250 4,200 3,500 3,000	5,730 4,584 3,820 3,274
8 Inches 10 Inches 12 Inches	1,910 1,528 1,273	2,387 1,910 1,592	2,600 2,100 1,750	2,865 2,292 1,910
14 Inches 16 Inches 18 Inches 20 Inches 22 Inches	1,091 955 849 764 694	1,364 1,194 1,061 955 868	1,500 1,300 1,150 1,050 050	1,637 1,432 1,273 1,146 1,042
24 Inches 26 Inches 28 Inches 30 Inches 32 Inches	637 586 546 509 477	976 733 683 637 596	875 800 750 700 650	955 879 819 764 716
34 Inches 36 Inches 38 Inches	449 424 402	561 531 503	620 580 550	674 637 603 573
40 Inches 42 Inches 44 Inches 46 Inches 48 Inches	382 364 347 332 318	478 455 434 415 397	525 500 475 4 5 5 440	546 521 498 477
50 Inches 52 Inches 54 Inches	306 294 283	383 369 354	420 405 390	459 441 425 410
56 Inches 58 Inches 60 Inches	273 264 255	341 330 319	375 360 350	396 383

The exact speed at which any specified wheel should be run depends upon several conditions, such as the type of machine, character of work and wheel, quality of finish desired and various other factors referred to at other places in this book. Wheels are ordinarily run in practice from about 4000 to 6000 feet per minute, though in some tases a speed as high as 7500 feet has been employed. An average

speed recommended by most wheel makers is 5000 feet. To allow an ample margin of safety it is recommended that wheel speeds should not exceed 6000 feet per minute.

The table of circumferences below will be of service in connection with the finding of surface speeds and spindle revolutions per minute.

CIRCUMFERENCES OF GRINDING WHEELS

Diameter	Circumference	Diameter	Circumference	Diameter	Circumference
of Wheel	of Wheel in	of Wheel	of Wheel in	of Wheel	of Wheel in
in Inches	Feet	in Inches	Feet	in Inches	Feet
1	.262	25	6.546	49	12.838
2	.524	26	6.807	50	13.090
3	.785	27	7.069	51	13.352
4	1.047	28	7.330	52	13.613
5	1.309	29	7.592	53	13.875
6 7 8 9	1.571 1.833 2.094 2.356 2.618	30 31 32 33 34	7.854 8.116 8.377 8.639 8.901	54 55 56 57 58	14.137 14.499 14.661 14.923 15.184
11	2.880	35	9.163	59	15.446
12	3.142	36	- 9.425	60	15.708
13	3.403	37	9.687	61	15.970
14	3.665	38	9.948	62	16.232
15	3.927	39	10.210	63	16.493
16 17 • 18 19	4.189 4.451 4.712 4.974 5.236	40 41 42 43 44	10.472 10.734 10.996 11.257 11.519	64 65 66 67 68	16.755 17.017 17.279 17.541 17.802
2 I	5.498	45	11.781	69	18.064
2 2	5.760	46	12.043	70	18.326
2 3	6.021	47	12.305	71	18.588
2 4	6.283	48	12.566	72	18.850

Thus, to find the surface speed of a wheel in feet per minute:

Rule. — Multiply the circumference as obtained from the table, by the number of revolutions per minute.

Example. — A wheel 18 inches diameter makes 1060 revolutions per minute. What is the surface speed, in feet, per minute?

 $4.712 \times 1060 = 5000$ feet surface speed.

When the surface speed and wheel diameter are given, to find the number of revolutions of the wheel spindle:

Rule. — Divide the surface speed in feet per minute by the circumference.

Example. — A wheel 24 inches diameter is to be run at 6000 feet surface speed per minute. How many revolutions should the wheel make? $6000 \div 6.283 = 962$, number of revolutions per minute the wheel

 $6000 \div 6.283 = 962$, number of revolutions per minute the whee should make.

Norton Co.

GRADING ABRASIVE WHEELS

THE Norton Company uses 26 grade marks, the Carborundum Company 19, while the Safety Emery Wheel Company uses 40. The following table is a comparison between the grade designations of the Norton Company and the Carborundum Company. Intermediate letters between the grade designations indicate relative degrees of hardness between them; the Norton Company manufacturing four degrees of each designation, while the Carborundum Company manufactures three.

Grade Designation

Norton Co.	Grade Designation	Carborundum Co.		
A B	Extremely or Very Soft	V		
C D E F	Soft	S		
$\begin{matrix} I \dots & H \dots & \dots & \dots \\ J \dots & J \dots & \dots & \dots \end{matrix}$	Medium Soft	Q		
K	Medium	N M L		
Q	Medium Hard	K		
S T V	Hard	Н G F		
Y Z	Extremely or Very Hard	D		

The Safety Emery Wheel Company's grade list is an arbitrary one with the following designations:

C. Extra Soft A. Soft H. Very Soft M. Medium Soft

I. Medium Hard

P. Medium

N. Very Hard

O. Hard E. Extra Hard

D. Special Extra Hard

Intermediate figures between those designated as soft, medium soft, etc., indicate so many degrees harder or softer, e.g., A_{4}^{1} is one degree harder than soft. A_{4}^{3} is three degrees harder than soft or one degree softer than medium soft.

NUMBERS AND GRADES OF ABRASIVE WHEELS

In the following table for the selection of grades will be found a comparison of the grading used by the Norton Company, and that of the Carborundum Company:

	I	Vorto	on Co.		Carborur	idum Co.
Class of Work		ally	Grade Usuall Furnish	y	Number Usually Furnished	Grade Usually Furnished
	-					
Large Cast Iron and Steel Castings	16 to	20	Q to 1	R	16 to 24	G to H
Small Cast Iron and Steel Castings	20 to	30	P to	Ç	20 to 30	G to H
Large Malleable Iron Castings	16 to	20	Q to 1	Ŕ		G to H
Small Malleable Iron Castings	20 to	30	P to (Ç	20 to 30	
Chilled Iron Castings	16 to	20	Q to I	Ř.	16 to 24	
Wrought Iron	16 to	30	P to 0	Ç	16 to 24	F to H
Brass Castings	16 to	30	O to	Ρ̈́	20 to 36	
Bronze Castings	r6 to	30	P to (Q.	20 to 30	I
Rough Work in General	16 to	30	P to (Ç	20 to 30	H
General Machine Shop Use	30 to	46	O to	P	24 to 36	G to T
Lathe and Planer Tools	30 to		N to (30 to 36	
Small Tools	36 to	100	N to I	Р	50 to 80	I to J
Wood-working Tools	36 to	60	M to I	V	40 to 60	
Twist Drills (Hand Grinding)	36 to	60	M to I	N	60	I to J
Twist Drills (Special Machines).	46 to	60	K to I	ΛI	50	L to O
Reamers, Taps, Milling Cutters,						
etc. (Hand Grind)	46 to	100	N to I	P	50 to 80	K to N
Reamers, Taps, Milling Cutters,						
etc. (Spec. Mach.)	46 to	60	H to I	Σ	50 to 60	L to M
Edging and Jointing Agricultural	•				Ĭ	
Implements	16 to	30	Q to I	2	141 to 24	G to I
Grinding Plow Points	16 to		P to (20 to 24	
Surfacing Plow Bodies	20 to		N to (16 to 20	
Stove Mounting	20 to	36	P to () l	24 to 30	G
Finishing Edges of Stoves	30 to		O to 1		24 to 30	
Drop Forgings	20 to	30	P to (ЭΙ	24 to 36	
Gumming and Sharpening Saws.	36 to		M to 1		403-603	I to L
Planing Mill and Paper Cutting	~				20260	•
Knives	30 to	46	J to E		to 80	M to R
Car Wheel Grinding	20 to	30	Ŏ to I	2	16 to 24	H
		_		1		

THE SHAPES OF WHEELS

Although grinding wheels are manufactured in a great variety of shapes and sizes, there are a few general forms into which they may be grouped. Practically all of the hundreds of commonly used shapes made for the various types of grinding machines and for the different kinds of work come under some one of these classifications, the most common of which may be designated as "disk," "cup," "cylin-

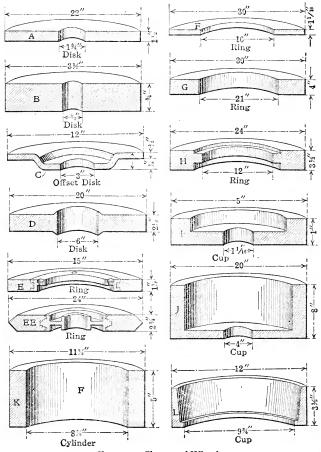


Fig. 9. — Shapes of Wheels

der" and "saucer" wheels. These shapes and some of their modifications are included in the groups of wheels illustrated in Figs. 9-10. These wheels are in most cases made in numerous widths and diameters, and the dimensions given in any such instances merely

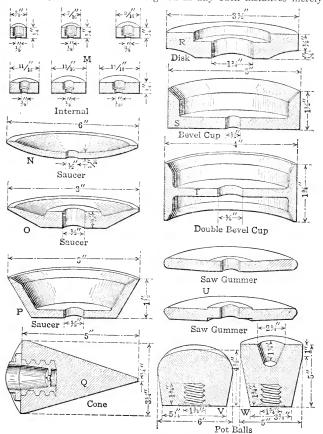


Fig. 10 — Shapes of Wheels

show the proportions of one size of wheel selected as typical from the comprehensive lists of wheel manufacturers' products.

Of the wheels shown, A and B are plain disks; C is an offset disk used on cylindrical grinders for grinding close up to a gear, collar, or piston head of large radius; D is a disk wheel for different makes of

tool and roll grinders, E, EE, F, G and H are "ring" wheels which are modifications of the disk type. The two wheels E and EE are made on large iron centers and are for use on the Sellers tool grinders. The other ring wheels referred to are adapted for mounting on large iron centers, and all such wheels are for service on machines where only a limited reduction of wheel diameter is permissible.

The shallow cup wheel I is for drill grinders and the larger, deeper cup wheel J is of a type used extensively on knife-grinding machinery. Wheels of this shape and of suitable dimensions, are also used on roll and cylindrical grinders, and in the smaller sizes on cutters and

reamer grinders.

The wheel K is a plain cylinder for grinding on the end the same as the cup wheels. Such cylinders are used in various proportions on vertical surface grinders, edge grinders, etc. They are held in ring chacker.

Cup and cylinder wheels are coming more and more into use and are already made in a great variety of diameters and widths. On certain classes of grinding operations they have marked advantages over the regular form of wheel as the same diameter is always maintained, thus avoiding the necessity of a change in speed, and the grinding is accomplished on a flat surface instead of one that is curved.

The cup wheel L is for the Pratt & Whitney vertical surface grinder. The small wheels in group M are for internal grinding operations in holes of limited diameter. The larger internal wheel R is for the Heald cylinder grinder. Three forms of the "saucer" wheel for cutter grinders are illustrated at N, O and P, and two bevel-edge cup wheels are shown at S, and T, both being made for cutter grinder use. The double-edge cup wheel can be used for sharpening both sides of a straddle mill without reversing the latter on its arbor; it is also useful in such operations as grinding out parallel surfaces, say the jaws of a snap gage.

The conical wheel at Q is for a tool grinder and is used principally for sharpening pattern makers' gouge's which are beyeled on the inside

of the curve.

Two saw gummers, representative of a number of shapes regularly made, are shown at U. These are adapted for sharpening saw teeth and grinding down in the "gullet" or concave space between the bottoms of the teeth.

The "pot balls" at N and W are used in grinding out hollow ware such as pots and kettles. These are made in great variety to suit

spherical receptacles, skillets, flat bottom pots, etc.

MOUNTING GRINDING WHEELS

One of the most important considerations in connection with the use of grinding wheels is that they shall be properly mounted, upon suitably proportioned spindles and between properly designed flanges. A wheel which is crowded upon a spindle of weak design, or which is cramped between two imperfect flanges that are either too small or take a bearing upon the wheel at the wrong point, is subjected to conditions as likely to cause an accident as is an excessive rate of speed.

The vast number of abrasive wheels in use upon the class of machines commonly known as bench and floor grinders, grinding wheel stands, emery grinders, etc., and which are so generally in service at various points about the machine shop, blacksmith shop and foundry, makes it desirable that something should be said here in reference to the best methods of mounting wheels on such apparatus.

In the first place, the machine itself should be of rigid construction, with spindle of ample proportions; the bearings should be well fitted; and kept well oiled so that the arbor will not become overheated and by expanding, break the wheel; and the machine should be securely fastened on substantial foundations not only to insure safety but in order to secure better results with the wheel.

The following sizes of spindles are recommended by the Norton Company and by some other wheelmakers, except where the grinding

wheels are extra thick:

Wheel										Spind	le
6 in. diameter and less									. 1/2	in.	
8 in. diameter and less									. 8	in.	
10 in. diameter and less											
12 in. diameter and less									. I	in.	
14 in. diameter and less											
16 in. diameter and less											
18 to 20 in. diameter											
22 to 24 in. diameter									. 2	in.	
Larger than 24 in									$.2\frac{1}{4}$	to 3	in.

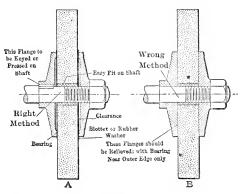


Fig. 11 — Right and wrong way to mount wheels

The flanges should be relieved as at A in Fig. 11, and they should be at least one-half the diameter of the wheel and have a true bearing at the outer edge. The inner flange should never be loose but in all cases should be fixed on the spindle. Under no circumstances should the flanges be allowed to be less than one-third the diameter

of the wheel. Wheels must not be allowed to run when held only by a nut in place of a flange, as the nut is liable to crawl and cause accident.

Compressible washers of pulp or rubber, slightly larger than the flanges, should be used between the flanges and the wheel as shown at A, Fig. 11. These distribute the pressure evenly when the flanges are tightened, by taking up any imperfections in the wheel or flanges.

The hole in the wheel bushing should be 0.005 inch larger than standard size spindles. This permits the wheel to slide on the spindle without cramping and insures a good fit not only on the spindle but against the inside flange, which is essential.

The flanges should be tightened only enough to hold the wheel

firmly, thus avoiding unnecessary strain.

General Suggestions

HANDLE all wheels with the greatest care in unpacking, storing, delivering, etc. Wheels are frequently cracked by rough usage before they are ever placed on the grinding machine. The man in charge of the storeroom should inspect each wheel before giving it out to the workman.

Wheels should be stored in a dry place.

A wheel used in wet grinding should not be left overnight partly

immersed in water.

When mounting wheels do not screw up the nut too tight; it should be set up only enough so that the flanges hold the wheel firmly.

Keep all rests adjusted close to the wheel so that work cannot be

caught

Avoid heavy pressure of the work on the wheel when grinding.

Keep the wheel true by dressing frequently.

If a wheel vibrates, there is something wrong. It should be trued up and the boxes should be rebabbitted after the journals are trued up.

Never hack a wheel; it is unnecessary and dangerous.

Use wheel guards wherever possible.

MAGNETIC CHUCKS

Magnetic chucks have come to be a very necessary part of the equipment of any surface grinding machine, whether plain or rotary. Before their coming it was customary to bed thin work in wax on the platen of a grinder in order to finish the flat sides. Other flat work had to be held in "fingers" on special fixtures, and on account of their being very thin and easily sprung, it was difficult to secure really accurate work.

The magnetic chuck holds the thinnest pieces of iron or steel firmly, draws down any slight spring in the work and prevents springing when

strains are released during the grinding operations.

The chuck face is divided into magnet poles, separated by babbit or other non-magnetic metals, and coils of insulated wire from these into electromagnets when current is applied. For rotary work, the electric current is supplied by brushes running against insulated contract rings on the outside. Current can be supplied from any incandescent lamp socket on a direct current circuit.

Alternating current cannot be used.

A No. o chuck having a face 10×14 inches uses about one-half as much power as a 16 candle-power lamp.

Hints for Using Magnetic Chucks

The chucks should not be taken apart.

Nothing but iron or steel can be held on the chucks.

The holding power depends on the amount of work surface in contact with the chuck.

Work can be held on edge by using adjustable back rest.

Very thin work can be held for grinding on the edges by laying it against the back rest and backing it up with a parallel strip.

Thin work will not hold as well as thick work.

In packing a number of small pieces on a chuck at one setting, it is better to separate them a little with strips of non-magnetic material.

Do not plug up the vent holes in the chuck.

Keep water away from the switch, the brushes and the interior of the chuck.

Magnetic chucks do not take the place of all other chucks.

Do not use water on chucks except where they are made for it. Chucks are usually wound for 110 or 220 volts for direct current only.

POLISHING WHEELS

THERE are many varieties of polishing wheels in use, the principal kinds being known as wooden wheels, compressed wheels, canvas and muslin, seahorse and felt wheels. For good work, and economy in abrasive, glue and labor cost, wheels and methods must be selected to suit the work.

A few years ago, the wooden wheel covered with leather and turned to fit the piece to be polished, was universally used. At the present time, the wooden, leather-covered wheel is used largely on flat surfaces and on work where it is necessary to maintain good edges. When this kind of a wheel is made with a double coating of leather, it makes a first-class finishing wheel.

Compressed wheels, or wheels having a steel center are made with surfaces of leather, canvas or linen. Many tool shops are equipped with these wheels exclusively. They answer all purposes and are safer and more economical than wooden wheels.

They are also used largely on cutlery and for polishing chilled plows. The compressed wheel is of strong construction, is very durable and

easily kept in balance.

Canvas and muslin wheels are used extensively for polishing stoves, shovels, plows, brass, cast iron and steel. For roughing out and dry fining on irregular pieces, they have proved very satisfactory. They hold the abrasive well and require no washing off as they can be cleaned with a buff stick or an abrasive brick.

Many concerns, such as plow-, shovel- and hoemakers, buy the canvas and muslin and make their own wheels.

Sea-horse wheels are very expensive, most concerns buying the

hides and making their own wheels.

Where a high-grade polish is required, there is probably no wheel which can compare with the wheel made of sea horse. They are largely used on guns, pistols and cutlery.

Felt wheels are largely used by stovemakers for finishing surfaces.

Bull neck wheels are also used for this purpose.

The felt wheels are made from white Spanish and Mexican felt and are extensively used for finishing on certain classes of work.

Care of Polishing Wheels

POLISHING wheels should be kept in perfect balance and running true at all times. A wheel out of balance wastes time, glue and abrasive, and will not do as good work.

The most efficient glue and the best abrasive are the cheapest in

the long run.

The glue pots should be kept clean and the glue properly cooked before using.

It is also important to heat the abrasive before applying.

The wheels should be kept properly cleaned and thoroughly

covered with the abrasive.

The wheels should be selected for the particular work the same as in grinding and only the wheel best adapted should be used at all times.

Polishing operations are usually divided into three classes: rough-

ing, dry fining, and finishing or oiling.

The abrasives used for roughing usually run from numbers 20 to 80. For dry fining, from numbers 90 to 120. The numbers used

for finishing run from 150 to XF.

For both roughing out and dry fining, the polishing wheels should be used dry. For finishing, the wheels are first worn down a little and then oil, beeswax, tallow and similar substances are used on the wheel. This, together with the abrasive, brings up a fine finish.

Speed of Buffing Wheels

Wood, leather covered	7,000 ft. per minute
Walrus	8,000 ft. per minute
Rag wheels	7,000 ft. per minute
Hair-brush wheels	12,000 ft. per minute
Ohio grindstone	
Huron grindstone	3,500 ft. per minute

LAPPING

LAPPING may be defined as the process of finishing the surface of a piece of work by means of another piece of material, called a lap,

the surface of which is charged with an abrasive.

Laps are roughly divided into three general classes. First, those where the form of the lap makes a line contact with the work, and the work is, if cylindrical, revolved to develop the cylindrical form, or, if straight, in one direction, is moved back and forth under the lap. Second, those which are used for straight surfaces with a full contact on the lap, and third, those which are used for male and female cylindrical surfaces with a full contact on the lap. In all cases the material from which the lap is made must be softer than the work. If this is not so, the abrasive will charge the work and cut the lap, instead of the lap cutting the work.

The first class is used in the place of emery wheels, either where the work is too small to use an ordinary wheel or where a form is to be ground on the work and an emery wheel will not keep its shape. They are usually made of machinery steel and the abrasive used is crushed diamond rolled into the surface. In rolling in the diamond

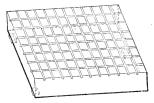


Fig. 1. — A Lapping Plate for Flat Work

dust the sharp corners of the particles cause them to bed securely into the surface of the lap, and if a good quality of diamond is used, a lap will grind all day without recharging. Oil is used to lubricate the work and carry away the dust from the grinding. If a diamond lap is run dry the particles of diamond tear and raise "burs" in the work, which strip the lap very quickly. The speed should be about two-thirds that for an emery wheel of the same size; for if it is excessive, the lap will wear smooth and glaze instead of cutting. This kind of lap is used mainly in watch and clock shops, and shops making watch tools, sub-press dies, and similar work.

Lapping Flat Surfaces

In lapping flat surfaces, which are usually on hardened steel, a cast-iron plate is used as a lap and emery as an abrasive. In order that the plate may stay reasonably straight, it should either be quite thick, or else ribbed sufficiently to make it rigid, and in any case it should be supported on three feet, the same as a surface place. For

rough work or "blocking down," as it is called, the lap works better if scored with narrow grooves, about $\frac{1}{2}$ inch apart, both lengthways and crossways, thus dividing the plate into small squares, as in Fig. The emery is sprinkled loosely on the block, wet with lard oil and the work rubbed on it; care is taken to press hardest on the highest spots. The emery and oil get in the grooves, and are continually rolling in and out, getting between the plate and the work and are crushed into the cast iron, thus charging it thoroughly in a short time. About No. 100 or No. 120 emery is best for this purpose.

After blocking down, or if the work has first been ground on a surface grinder, the process is different. A plain plate is used with the best quality of flour of emery as an abrasive, as the least lump or coarseness will scratch the work so that it will be very hard to get the scratches out. Instead of oil benzine is used as a lubricant and the lap should be cleaned off and fresh benzine and emery applied as often as it becomes sticky. The work should be tried from time to time with a straight-edge and care taken not to let the emery run in and out from under the work, as this will cause the edges to abrade more than the center, and will especially mar the corners. After getting a good surface, the plate and work should be cleaned perfectly dry, and then rubbed. The charging in the plate will cut just enough to remove whatever emery may have become charged in the work, will take away the dull surface and leave it as smooth as glass and as accurate as it is possible to produce.

Laps for Holes

In lapping holes various kinds of laps are used, according to the accuracy required, and the conditions under which the work is done. The simplest is a piece of wood turned cylindrical with a longitu-

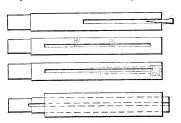


Fig. 2. — Laps for Holes

dinal groove or split in which the edge of a piece of emery cloth is inserted. This cloth is wound around the wood until it fills the hole in the work. This is only fit for smoothing or enlarging rough holes and usually leaves them more out of round and bell-mouthed than they were at first. Another lap used for the same purpose — and which produces better results — is made by turning a piece of copper, brass, or cast iron to fit the hole and splitting it longitudinally for some distance from the end. Loose emery is sprinkled over it,

with lard oil for a lubricant, and a taper wedge is driven into the

end for adjustment as the lap wears.

For lapping common drill bushings, cam rolls, etc., in large quantities, where a little bell-mouthing can be allowed, and yet a reasonably good hole is required, a great many shops use adjustable copper laps made with more care than the above. One way of making them is to split the lap nearly the whole length, but leaving both ends solid. One side is drilled and tapped for spreading screws for adjustment. Either one screw half-way down the split may be used or two screws dividing the split into thirds. Another and better means of adjustment is to drill a small longitudinal hole a little over half the length of the lap, enlarge it for half its length, and tap the large end for some distance. This is done before splitting. Into this hole a long screw with a taper point is fitted so that when tightened it tries to force itself into a small hole, thus spreading the lap.

For nice work there is nothing better than a lead lap. Lead charges easily, holds the emery firmly and does not scratch or score the work. It is easy to fit to the work and holds its shape well for light cuts. Under hard usage, however, it wears easily. For this reason, while laps for a single hole or a special job are sometimes cast on straight arbors, where much lapping is done it is customary to mold the laps to taper arbors with means for a slight adjustment. After any extensive adjustment the lap will be out of true and must be turned off. All of these laps, as shown in Fig. 2, are to be held by one end in a lathe chuck, and the work run back and forth on them by hand, or by means of a clamp held in the hand. If a clamp is used care

should be taken not to spring the work.

How to Do Good Lapping

There are several points which must be taken into consideration in order to get good results in lapping holes. The most important is that the lap shall always fill the hole. If this condition is not complied with the weight of the work and the impossibility of holding it exactly right will cause it to lap out of round, or if it is out of round at the start the lap will be free to follow the original surface. lap fits, it will bear hardest on the high spots and lap them off. in importance to getting a round hole is to have it straight. attain this end the lap should be a little longer than the work, so that it will lap the whole length of the hole at once, and not have a tendency to follow any curvature there may be in it. What is known as bell-mouthing, or lapping large on the ends, is hard to prevent. especially if the emery is sprinkled on the lap and the work shoved on it while it is running. The best way to avoid this condition when using cast-iron or copper laps, which do not charge easily, is to put the emery in the slot, near the center of the lap, and after the work is shoved on squirt oil in the slot to float the emery. Then, when the lathe is started the emery will carry around and gradually work out to the ends, lapping as it goes. Where lead is used the emery can be put on where it is desired to have the lap cut and rolled in with a flat strip of iron. It will not come out easily, so will not spread to any extent, and it is possible with a lap charged in this manner to avoid cutting the ends of the hole at all. The work should always be kept in motion back and forth to avoid lumping of the emery and cuttings which will score grooves in the work.

Ring Gage and Other Work

RING gages are lapped with a lead lap. They are first ground straight and smooth to within .0005 inch of size, and then, when lapped, are cooled as well as cleaned, before trying the plug, by

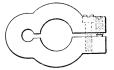


Fig. 3.—A Lap for Plugs

placing them in a pail of benzine for a long enough time to bring them down to the temperature of the room. Some shops leave a thin collar projection from each side around the hole, so that, if there is any bell-mouthing, it will be in these collars, which are ground

off after the lapping is done.

Other metals are lapped in this same manner, except that the abrasive is different. Cast iron is lapped with emery, but charges to some extent. This charging can be taken out without changing the work materially by rubbing it by hand with flour of emery cloth. In lapping bronze or brass, crocus and Vienna lime are used. Crocus is used with a cast-iron or lead lap, and the charging is removed by running the work for a few seconds on a hardwood stick which fits the hole. Unslaked Vienna lime, freshly crushed, is used with a lead or hardwood lap, and does not charge. It does a nice job, but is very slow, and is only used in watch factories.

For lapping plug gages, pistons, and other cylindrical articles, a cast-iron lap is usually used, split and fitted with a closing and a spreading screw, as shown in Fig. 3. Sometimes, where a very fine finish is required, or where the work is not hardened, the hole is

made larger than the work, and a lead ring cast into it.

DIAMOND POWDER IN THE MACHINE SHOP

The diamond used for this purpose, costing 85 cents per carat, is an inferior grade of diamond, not so hard as the black diamond used for drills and truing emery wheels, and not of a clear and perfect structure to permit it to enter the gem class. Many are a mixed black and white, others yellow and some pink; many are clear but flaky. Then there is the small débris from diamond cutting, which is reduced to powder and sells somewhat cheaper; but some find it more economical to use the above and powder it themselves, as the débris from diamond cutting is of a flaky nature, and does not charge into the lap so well.

Assuming there is 25 carats to reduce to powder, proceed as follows: Into a mortar, as shown at Fig. 4, place about 5 carats, using an 8-ounce hammer to crush it. It takes from 3 to 4 minutes' steady pounding to reduce it to a good average. Scrape the powder free from the bottom and the sides and empty into one half pint of oil. The oil used is the best olive oil obtainable, and is held in a cupshaped receptacle that will hold a pint and one half. The 25 carats being reduced to powder, and in the oil, stir it until thoroughly mixed, and allow to stand 5 minutes; then pour off to another dish. The diamond that remains in the dish is coarse and should be washed in benzine and allowed to dry, and should be repounded, unless extremely coarse diamond is desired. In that case label it No. o. Now stir that which has been poured from No. o, and allow to stand to minutes. Then pour off into another dish. The residue will be No. 1. Repeat the operation, following the table below.

The settlings can be put into small dishes for convenient use, enough oil staying with the diamond to give it the consistency of paste. The dishes can be obtained from a jewelers' supply nouse,

TABLE FOR SETTLING DIAMOND POWDER

To obtain No. 0 — 5 minutes.
To obtain No. 1 — 10 minutes.
To obtain No. 2 — 30 minutes.
To obtain No. 5 — 1 hour.
To obtain No. 4 — 2 hours.
To obtain No. 5 — 10 hours.
To obtain No. 5 — 10 hours.

Diamond is seldom hammered; it is generally rolled into the metal. For instance, several pieces of wire of various diameters charged with diamond may be desired for use in die work. Place the wire and a small portion of the diamond between two hardened surfaces, and under pressure roll back and forth until thoroughly charged. No. 2 diamond in this case is generally used. Or one can form the metal any desired shape and apply diamond and use a roll, as Fig. 6, to force the diamond into the metal. This is then a file which will work hard steel, but the moment this diamond file, or lap, is crowded it is stripped of the diamond, and is consequently of no use. It is to be used with comparatively light pressure.

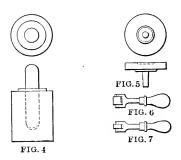
Diamond Laps

COPPER is the best metal. It takes the diamond readily, and retains it longer than other metals; brass next, then bessemer steel. The latter is used when it is wished to preserve a form that is often used.

For sharpening small, flat drills, say 0.008 to 0.100, a copper lap mounted on a taper shank, as in Fig. 5 and charged on the face with No. 2 diamond, using pressure on the roll, makes a most satisfactory method of sharpening drills. The diamond lasts for a long time if properly used, and there is no danger of drawing the temper on the drill. It is much quicker than any other method of sharpening.

To charge the lap use the roll, Fig. 7, supported on a T rest pressing firmly against the lap, being careful to have the roll on the center; otherwise instead of charging the lap it will be grinding the

roll. The diamond may be spread either on the lap or the roll, and the first charging usually takes twice the amount of diamond that subsequent charging takes. To avoid loss of diamond, wash the



Diamond Lap Tools

lap in a dish of benzine kept exclusively for that purpose. This can be reclaimed by burning the metal with acids, and the diamond can be resettled.

For the grinding of taper holes in hard spindles or for position work in hard plates, where holes are too small to allow the use of emery wheels, No. I diamond does the work beautifully. Or if it is wished to grind sapphire centers or plugs as stops, etc., a bessemer lap made in the form of a wheel and charged with diamond on the diameter does the work nicely.

Nos. 5 and 6 diamond are used on boxwood laps, mounted on taper plugs or chucks, and the diamond smeared on with the finger. The lap is run at high speed and used for fine and slow cutting

which also gives a high polish.

REAMER AND CUTTER GRINDING

Reamer Clearances

AFTER constant experimenting for a period of more than a year, the Cincinnati Milling Machine Company succeeded in establishing tables for four styles of reamers for obtaining what they consider to be the best clearances, the object being to grind clearances on reamers which would ream the greatest number of smooth holes with a minimum amount of wear. The four styles of reamers are as follows: Hand reamers for steel, hand reamers for gray iron and bronze, chucking reamers for gray iron and bronze, chucking reamers for steel. The company uses adjustable blade reamers almost exclusively, all of which are ground in the toolroom on their universal cutter and tool grinder. High speed steel reamers cling to nickel steel and do not cut it as well as carbon steel.

Fig. 1 is a cross-section of a hand reamer. Two clearance lines, A and B, are ground on the blades, a being the cutting clearance and b the second clearance called for in the table. The object of giving the adjustment for the second clearance so minutely is to provide a proper width of land, which equals .025 inch on all hand reamers for gray iron or bronze, and 0.005 inch on hand reamers for steel.

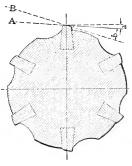


Fig. 1. - Cross-section of Hand Reamer

Chucking reamers for gray iron and bronze have, in this system, 23-degree beveled ends as shown in Fig. 2, and are provided with two clearances along the blades, for which the settings are given in Table 3. The beveled ends have only one clearance which is equal to the second clearance given in Table 3. Fig. 3 shows a chucking reamer for reaming steel. In these reamers the blades are circular ground to the exact size of hole to be reamed and without clearance, the 45-

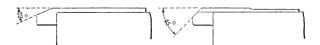


Fig. 2. — Chucking Reamer Blade for Gray Iron and Bronze

Fig. 3.—Chucking Reamer Blade for Steel

degree beveled ends only having clearance as given in Table 4. On all reamers of this style the blades are ground from .o15 to .o20 inch below size half of their length toward the shank end.

In grinding the clearances for the various kinds of reamers as given in Table 1, 2, and 3, the tooth rest is held stationary on the emery wheel head of the grinder, while in grinding the 45-degree beveled ends on the chucking reamers for steel, the tooth rest is supported from the grinder table and travels with the work. The front end of the hand reamer blades are tapered about 0.004 per inch. The back ends of the blades are also slightly tapered to prevent injuring the holes when backing the reamer out.

Tooth Set Rest REAMER CLEARANCE Below Work Ground with Cup Wheel 3" dia .- Tooth Rest to be Set Central Holding Centers. with Emery Wheel Spindle. Set Work holding Centers above Amount given Table Emery Wheel Center by Amount given below in Tables No. Below in No. 4 1-2 and 3 TABLE I TABLE 2 TABLE 3 TABLE 4 Hand Hand Reamer for Steel Reamer Chucking Chucking Ream-Cut'g Clearance Land for Cast Iron Reamer for ers for Steel Cirand Bronze Cut'g Clear-Cast Iron and cular Ground Bronze Cut'g Clearance ance Land .025 Wide Land .025 Wide For Cutting Clearance For Second Clearance For Sec... For Cutting Clearance For Cutting Clearance For Cutting Clearance For Second Clearance d 00 Reamer n g le E n d Blade Size ⋖ 129.4 15" " .080 45 Degrees .080 .012 .052 .032 .072 .040 " .057 .072 .080 .080 .012 .032 .040 45 .. .012 .062 .032 .072 .040 .000 45 .000 .. .012 .067 .035 .005 .040 .100 45 .100 . . .072 .005 .IOO .100 .012 .035 .010 45 46 .077 .005 .045 .125 .125 .012 .037 45 .. .082 45 .012 .040 .120 .045 .125 .125 .. .087 .120 .012 .040 .045 .125 45 .125 . . .012 .092 .040 .120 .045 .125 45 .125 .120 .012 .097 .040 .045 .125 45 .125 .012 .120 .102 .040 .045 .125 45 .125 .106 .122 .012 .042 .125 .125 .045 45 .012 .112 .145 .160 .160 .045 .050 45 " .012 .118 .045 .145 .050 .160 45 .160 .. .012 .122 .145 .050 .160 .175 .045 45 .. .045 .145 .012 .127 .055 .175 45 .175 .. .012 .048 .168 .175 ·I32 .055 .175 45 .. .012 .170 .055 .175 .I 37 .050 .175 45 .. .012 .142 .050 .170 .060 .200 45 .200 .. .060 .200 .012 .147 .050 .170 .200 45 .. .012 .060 .200 .152 .052 .102 .200 45 .060 .200 .012 .157 .192 .200 45 .052 .162 .196 .060 .056 .200 .200 OT2 45 .064 .012 .167 .056 .196 .200 45 ,200 " .. .064 .225 2 .012 .172 .056 .216 .224 45 44 .012 .172 .056 .216 .064 .224 45 .225 44 .012 .172 .059 .210 .064 .224 45 .225 .064 44 .012 .172 .050 .210 .224 45 .225 44 .012 .172 .063 .223 .064 .224 45 .225 ٤, .064 .012 .172 .063 .223 .224 45 .225 .. .012 .172 .063 .223 .068 .228 45 .230 2 1 7 " 2 1 7 " 2 2 2 " " .068 .228 .230 .012 .063 45 .172 .223 " .072 .012 .172 .065 .225 .232 45 .230 2 1 6 " 2 5 " 2 1 1 1 " 2 1 1 1 " 2 1 1 3 1 " 66

.065 Mount Tooth Rest on Emery Wheel Head

.065

.065

.065

.012

.012

.012

.012

.172

.172

.172

.172

.072

.075

.075

.077

.225

.225

.225

.225

.232

.235 45

.235 45

.237 45

45 "

Mount Tooth Rest on Table of Machine

"

.230

.235

.235

.240

REAMER CLEARANCE

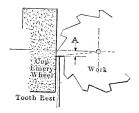
Ground with Cup Wheel 3" dia.—Tooth Rest to be Set Central with Emery Wheel Spindle. Set Work holding Centers above Emery Wheel Center by Amount given below in Tables No. 1-2 and 3

Set Tooth Rest
Below Work
Holding Centers.
Amount given
Below in Table
No. 4

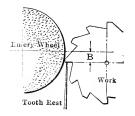
	TABLE 1 Reamer f Clearand Vide		Hand for C and Cut'g ance	Reamer ast Iron Bronze Clear- Land	Chuc Ream Cast I Bronz Clea	king eking er for Ironand ze Cut'g irance d .025	Chucking ers for S cular Gi	Ream- teel Cir-
Size of Reamer	For Cutting Clearance	For Second Clearance	For Cutting Clearance	For Second Clearance	For Cutting Clearance	For Second Clearance	Angle on End of Blade	For Cutting Clearance on Angle
######################################	.012 .012 .012 .012 .012 .012 .012 .012	.172 .172 .172 .172 .172 .172 .172 .172	.065 .070 .070 .070 .072 .072 .075 .078 .081 .081 .084 .084 .087 .090 .090 .090 .090 .090 .096 .096 .096	.225 .230 .232 .232 .233 .235 .235 .241 .244 .247 .247 .247 .247 .245 .250 .250 .256 .256 .256 .256 .256 .256 .256 .256	.077 .085 .085 .085 .085 .083 .083 .083 .083 .097 .090 .090 .090 .093 .097 .100 .104 .104 .104 .106 .106 .108 .108 .108 .108 .108 .108 .111 .114 .116 .116 .116	.237 .240 .240 .240 .240 .243 .243 .247 .250 .253 .253 .253 .257 .260 .264 .264 .264 .266 .268 .268 .268 .268 .268 .268 .268	45 Degrees 45 " 45 " 45 " 45 " 45 " 45 " 45 " 45 "	.240 .240 .240 .240 .240 .240 .245 .245 .245 .250 .250 .255 .255 .255 .266 .266 .266 .265 .265

Mount Tooth Rest on Emery Wheel Head

Mount Tooth Rest on Table of Machine



the table below.



CUP WHEEL CLEARANCE TABLE DISK WHEEL CLEARANCE TABLE

For setting tooth rest to obtain 5° or 7° clearance when grinding peripheral teeth of milling cutters with cup-shaped wheel. Tooth rest is set below work centers as at A, the distance being found in when grinding with periphery of disk wheel.

Dia. Cutter Inches	For 5° Clearance A =	For 7° Clearance A =	Dia. of Emery Wheel Inches	For 5° Clearance B =	For 7° Clearance B =
14 38 14 55 8 34	.011 .015 .022 .028	.015 .022 .036 .037	2 2 ¹ / ₄ 2 ¹ / ₂ 2 ³ / ₄ 3	.0937 .099 .110 .125	.125 .141 .156 .172 .187
78 I I 14 I 12 I 34	.037 .044 .055 .066	.052 .060 .075 .090	3 ¹ / ₄ 3 ¹ / ₂ 3 ³ / ₄ 4 4 ¹ / ₄	.143 .154 .165 .176	.203 .219 .234 .250 .265
$ \begin{array}{c} 2 \\ 2\frac{1}{4} \\ 2\frac{1}{2} \\ 2\frac{3}{4} \\ 3 \end{array} $.088 .099 .110 .121	.120 .135 .150 .165 .180	4½ 4¾ 5 5½ 5½	.198 .209 .220 .231	.281 .297 .312 .328
3½ 4 4½ 5 5 5 6	.154 .176 .198 .220 .242 .264	.210 .240 .270 .300 .330 .360	5 ³ / ₄ 6 6 ¹ / ₄ 6 ¹ / ₂ 6 ³ / ₄ 7	.253 .264 .275 .286 .297 .308	·359 ·375 ·390 ·406 ·421 ·437

OILSTONES AND THEIR USES

Natural Stones

THE following particulars regarding the well-known Arkansas and Washita stones are given by the Pike Manufacturing Company:

Arkansas stones are made from rock quarried in the Ozark mountains of Arkansas, and are prepared for commercial purposes in two

grades, hard and soft.

Hard Arkansas is composed of pure silica and its sharpening qualities are due to small, sharp-pointed grains, or crystals, of hexagonal shape, which are much harder than steel and will, therefore, cut away and sharpen steel tools. The extreme fineness of texture makes this stone, of necessity, a slow cutter, but in the very density of the crystals of which it is composed lies its virtue as a sharpener.

Soft Arkansas stone is not quite so fine-grained and hard as the hard Arkansas, but it cuts faster and is better for some kinds of mechanical work. It is especially adapted for sharpening the tools used by wood carvers, file makers, pattern makers, and all workers

in hard wood.

Washita stone is also found in the Ozark mountains in Arkansas and is similar to the Arkansas stone, being composed of nearly pure silica, but is much more porous. It is known as the best natural stone for sharpening carpenters' and general wood workers' tools. This stone is found in various grades, from perfectly crystallized and porous grit to vitreous flint and hard sandstone. The sharpness of the grit depends entirely upon its crystallization, the best oilstones being made from very porous crystals.

In addition to the regular rectangular sections, natural stones are made in such shapes as square, triangular, round, flat, bevel, dia-

mond, oval, pointed. knife edge.

Artificial Oilstones

ARTIFICIAL oilstones are manufactured in a multitude of shapes and sizes and are adapted for sharpening all kinds of tools. Such stones are made by the Norton Company of alundum and crystolon, the former being known as India oilstones, the latter as crystolon sharpening stones. Similar shapes are manufactured by the American Emery Wheel Works, and the Carborundum Company also makes such stones in great variety.

The stones are made in three grades or grits, coarse, medium and fine. The coarse stones are used in machine shops for sharpening very dull or nicked tools, machine knives, and for general use where

fast cutting is desired.

Medium stones are sharpening mechanics' tools in general, more particularly those used by carpenters and in wood-working shops. Fine stones are adapted for engravers, die workers, cabinet makers

and other users of tools requiring a very fine, keen-cutting edge.

Of the great variety of shapes and sizes a number adapted especially for machine shop purposes are illustrated half size in Fig. 1. Of these, Nos. 0, 1, 1½, 2, 24 and 29 are for sharpening lathe and planer

tools, and for use after grinding; Nos. 23, 25, 56 and $56\frac{1}{2}$ for reamers; Nos. 13, 14 and 15 for taps; Nos. 4, 5, 6, 7, 8, 9, 10, 11, 12 and 26 for dies.

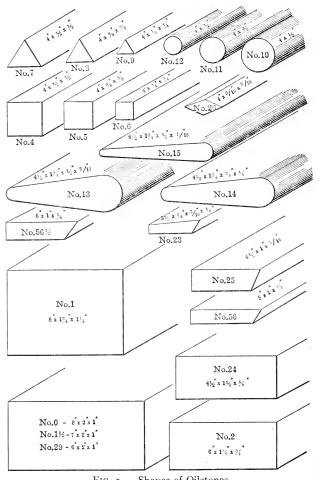


Fig. 1. - Shapes of Oilstones

A few shapes and sizes for curved-edge wood-working tools are Nos. 10, 11, 12, 13, 14, 15, in Fig. 1. Rectangular shapes for straightedge

tools like chisels, plane bits, planer knives, scrapers, paper-cutting knives and other tools with broad flat edges are Nos. 0, 1, $1\frac{1}{2}$, 2, 24, 29, in Fig. 1.

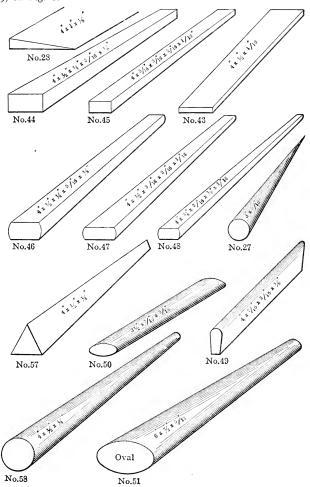


Fig. 2. — Shapes of Oilstones

Fig. 2 shows some of the sizes and shapes particularly adapted for mold and die work, watch and clock makers' tools, etc. Other

shapes suitable for such purposes are shown by Nos. 4, 5, 6, 7, 8, 9 and 26 in Fig. 1 and No. 51 in Fig. 2.

How to Care for Oilstones

LIKE anything else, an oilstone can be ruined by wrong treatment and lack of care.

There are three objects to be attained in taking good care of an oilstone: first, to retain the original life and sharpness of its grit; second, to keep its surface flat and even; third, to prevent its glazing.

To retain the original freshness of the stone, it should be kept clean and moist. To let an oilstone remain dry a long time, or expose it to the air, tends to harden it. A new natural stone should be soaked in oil for several days before using. If an oilstone is kept in a dry place (most of them are) it should be kept in a box with closed cover, and a few drops of fresh, clean oil left on it.

To keep the surface of an oilstone flat and even simply requires care in using. Tools should be sharpened on the edge of a stone as well as in the middle to prevent wearing down unevenly, and the

stone should be turned end for end occasionally.

To restore an even, flat surface grind the oilstone on the side of a

grindstone or rub it down with sandstone or an emery brick.

To prevent an oilstone from glazing requires merely the proper

use of oil or water.

The purpose of using either oil or water on a sharpening stone is to float the particles of steel that are cut away from the tool, thus preventing them from filling in between the crystals and causing the stone to glaze.

All coarse-grained natural stones should be used with water. Use

plenty of it.

On medium and fine-grained natural stones and in all artificial stones, oil should be used always, as water is not thick enough to

keep the steel out of the pores.

To further prevent glazing, the dirty oil should always be wiped off the stone thoroughly as soon as possible after using it. This is very important, for if left on the stone, the oil dries in, carrying the steel dust with it. Cotton waste is one of the best things to clean a stone with, and is nearly always to be found in a shop.

If the stone does become glazed or gummed up, a good cleaning with gasolene or ammonia will usually restore its cutting qualities, but if it does not, then scour the stone with loose emery or sandpaper

fastened to a perfectly smooth board.

Never use turpentine on an oilstone for any purpose.

SCREW MACHINE TOOLS, SPEEDS AND FEEDS

BOX TOOLS AND CUTTERS

THE general principles of two types of box tools using respectively tangent and radial cutters are represented in Figs. 1 and 2. The former type is generally used for roughing and the latter for finishing. The tangent cutter in the type of box tool shown in Fig. 1 lies in a slot formed parallel to the bottom of the box but at an angle, usually

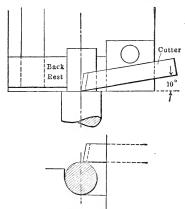


Fig. 1.—Roughing Box Tool with Tangent Cutter

ten degrees, with the front of the box, thus giving the desired rake at the cutting point. Finishing cutters of the type in Fig. 2 are straight on the end, located square with the work and ordinarily ground as indicated to give 7 to 10 degrees front clearance for steel

and 5 to 8 degrees for brass.

The tangent cutter is sharpened by grinding on the end, and compensation for the grinding away of the metal is made by adjusting the cutter forward, whereas in the radial type of cutter in Fig. 2, frequent sharpening cannot be done without resulting in lowering the cutting edge of the tool below the center of the work, unless a substantial part of the tool be sacrificed. The radial tool, however, is easily ground accurately on face a, which is the edge governing the finish; while the corresponding face on the tangent tool is rather difficult to grind so as to produce as smooth work.

The sizes of steel recommended for box-tool cutters are as follows: For box tools used for stock diameters up to $\frac{1}{16}$ inch, $\frac{3}{16}$ inch square; up to $\frac{3}{8}$ inch diameter, $\frac{7}{32}$ inch square; up to $\frac{3}{2}$ inch diameter, $\frac{1}{16}$ inch square; up to $\frac{3}{4}$ inch diameter, $\frac{1}{16}$ inch square; up to $\frac{1}{2}$ inches diameter, $\frac{1}{2}$ inch square.

HOLLOW MILLS

The teeth of hollow mills should be radial or ahead of the center. With the cutting edge ahead of the center, as in Fig. 3, the chips as produced are caused to move outward away from the work and prevented from disfiguring it. With the cutting edge below the center, rough turning will result. With the cutting edge greatly above the center, chattering occurs. About one tenth of the cutting diameter is a good average amount to cut the teeth ahead of the center.

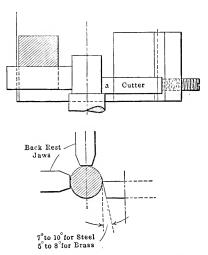


Fig. 2. — Finishing Box Tool with Radial Cutter

When the chips produced from any turning or boring cut curl nicely, it is indicative of a free cutting action; but these chips are very troublesome on the automatic screw machine. In making hollow mills for the automatic, part or all of the rake to the cutting edge is generally sacrificed.

The table under the hollow mill in Fig. 3 gives proportions of mills from $\frac{1}{16}$ to $\frac{3}{4}$ diameter, showing the amount to cut the teeth

ahead of the center, the taper of the hole, etc.

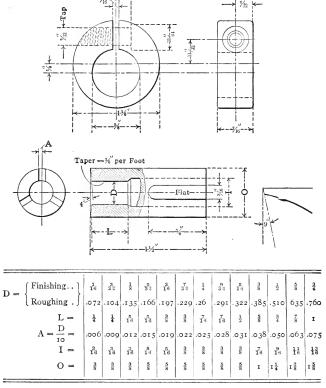


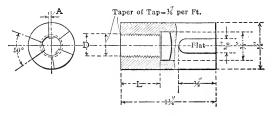
Fig. 3. - Hollow Mill Dimensions

DIES AND TAPS

It is good practice in making spring screw dies to either hob out the thread with a hob tap 0.005 to 0.015 inch over-size, according to size, and in use to spring the prongs to proper cutting size by a clamping ring, or to tap the die out from the rear with a hob tap tapering from $\frac{3}{16}$ inch to $\frac{1}{4}$ inch per foot, leaving the front end about 0.002 inch over cutting size, and in this case also to use a clamping ring. Both of these schemes are for the purpose of obtaining back clearance and are effective. Of the two the use of the taper hob is to be preferred.

Spring Die Sizes

The table of dimensions for spring screw dies, Fig. 4, should prove of service, particularly for steel. For brass the cutting edge is radial, thus eliminating dimension A. The width of land at bottom of thread is usually made about $\frac{1}{4}$ outside diameter of cut, the milling between flutes being 70 degrees, leaving 50 degrees for the prong in the case of three-flute dies.



SMALL SIZES OF DIES (Over all Dimensions Given in Sketch)

D =	16	38	3 16	14	15 16	No.	4	6	8	10	12
Threads P. I. =	64	40	32	20	18	56	40	32	24 - 32	24 - 32	24- 32
$A = \frac{D}{10}$.003	.012	.019	.025	.031	.010	.011	.014	.016	0.19	.021
L =	3 16	$\frac{9}{32}$	3 8	$\frac{1}{2}$	$\frac{1}{2}$	$\frac{7}{32}$	14	$\frac{9}{32}$	16 16	$\frac{1}{3}\frac{1}{2}$	38

Sizes 3 to 1 Inch

		1	
D = Th' s P.I. =	$\frac{3}{8}$ to $\frac{1}{2}$ Std.	$\frac{1}{2}$ to $\frac{3}{4}$ Std.	₹ to 1 Std.
A =	D ÷ 10	$D \div 10$	D ÷ 10
L = 1	34	ı"	11/2
O.S. Dia.	1"	I 1/8	15
Length	2"	21/	$2\frac{1}{2}''$

Fig. 4. — Spring Die Dimensions

Sizing Work for Threading

In boring holes previously to tapping they should be somewhat larger than the theoretical diameter at bottom of thread, as the crowding action of the tap will cause the metal to flow some and compensate for this. Where no allowance is made, frequent tap breakage is liable to occur and torn threads in the work also. On

external work it is for the same reasons advisable to turn the work undersize and the following table gives good average allowances for both internal and external work.

Allowances for Threading in the Screw Machine

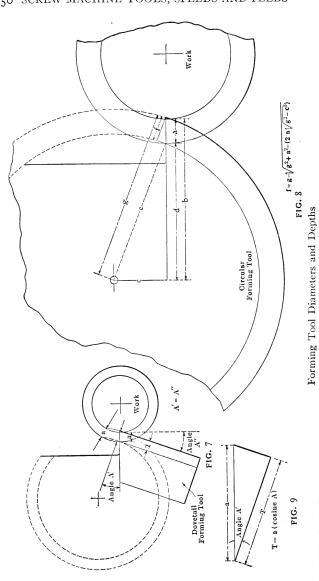
Threads per Inch	External Work Turn Undersize	Internal Work Increase Over Theoretical Bottom of Thread
28	0.002	0.004
24	0.002	0.0045
22	0.0025	0.005
20	0.0025	0.0055
16	0.003	0.006
14	0.003	0.0065
13	0.0035	0.007
· I2	0.0035	0.007
II	0.0035	0.0075
10	0.004	0.008
9	0.004	0.0085
8	0.0045	0.009
7 6	0.0045	0.0095
6	0.005	0.010

Tap Length and Number of Lands

The number of teeth in taps and the width of land should be regulated by the diameter and pitch of work as well as the nature of the material being cut. On fine threads, where a drunken thread is to be insured against, more teeth are required than on a coarser pitch of the same diameter. A good average number of teeth on taps for United States standard threads is given in the following table. With

Outside Dia.	No. of Flutes	Width of Land
$\frac{3}{1,6}$	4	$\frac{3}{6}$ 4
1 4 5	4	16
16	4	64
3 8	4	$\frac{3}{32}$
$\frac{7}{1.6}$	4	57 6 4
$\frac{1}{2}$	4	18
5 8 3	4	$\frac{5}{32}$
3 4	4	$\frac{3}{16}$
78	4	$\frac{7}{32}$
I	4	1/4
1 1	4	T 6

too few teeth and too short land very little support is afforded and this may cause chattering; too much land in contact causes heat due to excessive friction, welding of chips and torn threads.



FORMING TOOLS

The two types of forming cutters commonly used in the screw machine are shown in Figs. 5 and 6. The circular forming cutter in Fig. 5 is usually cut away from $\frac{1}{8}$ to $\frac{3}{18}$ 5, inch below center to give suitable cutting clearance and the center of the tool post on which it is mounted is a corresponding amount above the center of the machine, so that the cutting edge of the circular tool is brought on

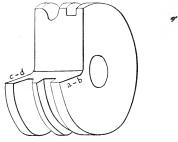


Fig. 5. — Circular Forming Tool

the center line of the work. The relative clearance ordinarily obtained by circular cutters and dovetail tools of the type shown in Fig. 6, is indicated in Fig. 7. It is obvious that with a given material the larger the diameter of the work the greater the angle of clearance required. Clearance angles are seldom less than 7 degrees or over 12 degrees.

The diameter of circular forming tools is an important matter for consideration. A small diameter has a more pronounced change of

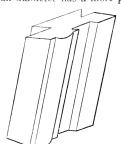


Fig. 6. - Dovetail Forming Tool

clearance angle than a large diameter. In fact, when of an exceedingly large diameter the circular tool approaches in cutting action the dovetail type of tool which is usually provided with about 10 degrees clearance. Circular tools usually range from about 13 inches diameter, depending upon the size of machine in which they are used.

Getting the Tool Diameters at Different Points

In order to make a circular or a dovetail type of tool so that the contour of its cutting edge is such as to produce correct work, the amount a circular tool is cut below center, as at c in Fig. 8, and the clearance angle of a dovetail tool as at A', Fig. 7 must be known. Thus, referring to Fig. 8, the forming tool shown cuts two different diameters on the work, the step between being represented by dimension a. To find depth f to which the forming tool must be finished on the center line to give the correct depth of cut a in the work (the

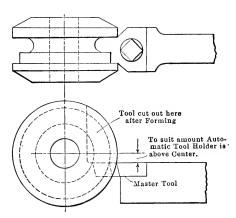


Fig. 10. - Finishing a Circular Tool

cutter being milled below center an amount represented by c) the following formula may be applied:

$$f = g - \sqrt{g^2 + a^2 - (2a\sqrt{g^2 - c^2})}$$
.

Suppose the depth of cut in the work represented by a to be 0.152"; the radius g of the forming cutter r inch; the distance c which the forming tool is milled below center, $\frac{3}{16}$ inch. Applying the above formula to find f and substituting the values just given for the letters in the formula we have $f = r - \sqrt{r + .0231 - (.304 \sqrt{r} - .03516)}$

$$= I - \sqrt{I + .023I - (.304 \times .9823)}$$

$$= I - \sqrt{.724485} = I - .8512 = .1488$$

Then f = .1488

Dovetail Tool Depths

If a similar piece of work is to be formed with a dovetail type of cutter, the distance T, Figs. 7 and 9, to which it is necessary to plane the tool shoulder in order that it may cut depth a correctly in the work, is found by the formula: T=a (cosine A'). As 10 degrees is the customary clearance on this form of tool, the cosine of this angle, which is .98481, may be considered a constant, making reference to a table of cosines unnecessary as a rule. Assuming the same depth for a as in the previous case, that is .152 inch, and multiplying by .98481, gives .1496 inch as the depth of T to which the tool must be planed.

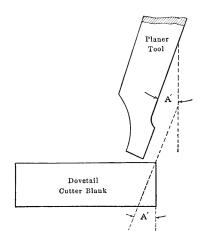


Fig. 11. - Finishing a Dovetail Forming Tool

While it frequently is necessary or advisable to determine by calculation the dimension computed in the preceding examples, in the majority of cases when making a cutter with a master tool of the same outline as the model, the correct form in the circular cutter is obtained automatically by dropping the master tool to the same distance below the lathe center as the circular cutter is to be milled off center and then feeding it in to finish the cutter. This procedure, shown in Fig. 10, assures the correct shape at all points being produced on the exact working plane of the cutter. Similarly, in finishing a dovetail cutter in the planer or shaper, the master tool may be set as in Fig. 11 at the same angle with the cutter (usually 10 degrees) as the latter will afterward be applied to the work.

CIRCULAR TOOL FOR CONICAL POINTS

When a circular cutter is to be made for forming a conical surface on a piece as in Fig. 12, a master tool of the exact angle required on the work may be used for finishing the cutter in the same way as the tool in Fig. 9 is applied; that is, the master is to be dropped below center the amount the cutter center is to be above the work center when in operation. The distance is represented by A in Fig. 12. Another method, which avoids the necessity of making a master tool, is to set the compound rest of the lathe to the exact angle required (in this case 30 degrees with the center line) and with a horizontal

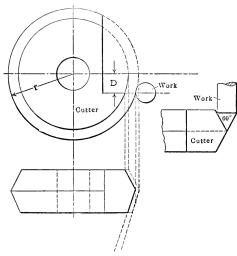


Fig. 12. — Circular Forming Tool for Conical Points

cutting tool set at distance D below center, turn off one side of the cutter blank and then set the compound rest around the other way and face off the other side. If desired a similar method may be followed for grinding the forming cutter after hardening. The arbor carrying the cutter should be located either above or below the grinding wheel a distance equal to $D\frac{(R+r)}{r}$, where D equals the depth the cutter is milled below center, r the radius of the cutter, and R the radius of the emery wheel. Assuming D to be .187 $(\frac{3}{16})$ inch; R, 2.5 inches; and r, 1 inch, the vertical distance between centers of forming tool and grinding wheel centers would equal .187 $(\frac{1+2.5}{r})$

= .187 (3.5) = .6562 $(\frac{21}{32})$ inch.

FINDING DIAMETERS OF CIRCULAR FORMING TOOLS

In making circular forming tools it is oftentimes desirable to check the finished tool or finish a tool by grinding. It may also be advantageous to know the exact diameter a tool should be turned while making it, in order that calipering may be more converient accretain. Methods of computing the diameter at different points are given on page 252, but in many cases of this kind the following tables will greatly facilitate matters, particularly when making

circular forming tools for Brown & Sharpe automatic

screw machines:

Suppose, for example, we have a piece to make like Fig. 13, on the No. 2 Brown & Sharpe automatic screw machine. The largest diameter of the circular forming tool would produce the smallest

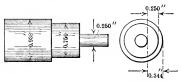


Fig. 13. — The piece to be made

diameter of the piece, which is 0.250 inch. The difference between this 0.250 inch diameter and the step of the 0.750 inch diameter

is $(0.750 - 0.250 \text{ inch}) \div 2 = 0.250 \text{ inch}$.

The largest diameter of the circular forming tool for the No. 2 machine is 3 inches, which corresponds to a radius of 1.49998 inches with a base line of 1.479 inch for the triangle completed by the perpendicular joining the cutting line of the tool with the parallel line passing through the center of the tool.

The hypothenuse of the triangle is formed by the radius joining the intersection of the base line and the circumference of the tool as in Fig. 14. Subtracting 0.250 inch from 1.470 inches we have 1.229 inches, which, in Table 4, corresponds to a radius of 1.25417 inches, and multiplying by 2 gives a diameter of 2.50834 inches, to which to turn the cutter to correctly form the 0.750-inch diameter on the piece, Fig. 13.

Considering the largest diameter of the piece and taking the height of the second step above the first diameter, we have $\frac{0.938 - 0.25}{0.938} = \frac{0.938 - 0.25}{0.938}$

0.344 inch, and subtracting from 1.479 = 1.135 for the base line, which, in Table 4, corresponds to a radius of 1.16221. Multiplying by 2 gives a diameter of 2.3244 to turn the cutter to in order to produce the 0.938-inch diameter on the work. Tables 2, 3 and 4 are for cutters of the dimensions given in Table 1.

These tables are figured in steps of 0.001 inch for the capacity of the machines. A difference of a fractional part of a thousandth can be added to the radius if the step is a part of a thousandth over the base-line figures, which are given in even thousandths. For illustration: Say the base-line figure is 1.4765 inches. In the table 1.476 inches corresponds to a radius of 1.49702; add 0.0005 inch to 1.49702 and the radius will be as near correct as it is practicable to make a cutter.

256 SCREW MACHINE TOOLS, SPEEDS AND FEEDS

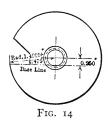


Table 1. — Dimensions of Cutters for B. & S. Automatic Screw Machines

Mach.	Approx.	Max.	Max.	Distance Above
No.	Diam.	Radius	Base Line	or Below Center
2	3"	1.49998"	1.479"	0.250"
0	21"	1.12474"	1.109"	0.1875"
00	13"	0.87497"	0.866"	0.125"

Table 2.—For Finding Diameters of Circular Forming Tools for Brown & Sharpe No. 00 Automatic Screw Machine

N	0. 00	No	0.00	N	0. 00	N	0. 00	No. 00						
Base Line	Radius	Base Line	Radius	Base Line	Radius	Base Line	Radius	Base Line	Radius					
.866 .865 .864 .863 .860 .858 .857 .855 .855 .855 .855 .855 .855	.87497 .87308 .87300 .87202 .87003 .86004 .86605 .86607 .86508 .86409 .86310 .86112 .86011 .85114 .85815 .8514 .85518 .85518 .85518 .85518 .85518 .85518 .85518 .85518	.841 .840 .839 .838 .837 .836 .833 .833 .833 .833 .832 .828 .827 .826 .825 .821 .822 .821 .820 .818	.85024 .84925 .84925 .84628 .84529 .84529 .84133 .84134 .84035 .8338 .83639 .83542 .83544 .83145 .83145 .83145 .83145 .83244 .83145 .83244 .83145 .83246 .83	.815 .814 .813 .812 .810 .808 .807 .806 .805 .804 .803 .802 .801 .800 .798 .707 .795 .795 .794 .792 .791	.82453 .82354 .82255 .82158 .81959 .81860 .81761 .81465 .81465 .81267 .81168 .81166 .80877 .80877 .80773 .80577 .80378 .80270 .80270 .80270	.789 .788 .787 .786 .785 .784 .783 .781 .780 .777 .776 .777 .776 .773 .771 .776 .769 .766 .766 .766 .766	.79884 .79785 .79686 .79389 .79390 .79390 .79291 .78995 .78995 .78569 .78502 .78502 .78404 .78205 .78308 .78508 .787107 .78107 .78107 .78107 .77515	.763 .762 .761 .760 .750 .755 .755 .755 .755 .755 .755 .75	-77317 -77218 -77120 -77021 -76022 -76823 -76725 -76528 -76528 -76429 -76330 -76232 -76133 -76033 -75936 -75837 -75739 -75739 -75544 -75344 -75344 -75344 -75344 -75344 -75344					

Note. — In the above Table 2 and Tables 3 and 4, it should be noted, as explained on page 255, that the Base Line dimensions in the columns under that heading and in the diagram, Fig. 14, are actually the distance of the cutting edge of the tool from the center of the cutter, the latter being used in the machine a certain distance either above or below the spindle center corresponding to Table 1. The distance from the cutting edge to center is therefore shorter than the true cutter radius by the amount indicated in the tables.

Table 3.—For Finding Diameters of Circular Forming Tools for Brown & Sharpe No. o Automatic Screw Machine

N	Го. о	1	Vo. 0	1	No. 0	N	0. 0	N	0.0
Base Line	Radius	Base Line	Radius	Base Line	Radius	Base Line	Radius	Base Line	Radius
1.109 1.108 1.107 1.106 1.105 1.104 1.103 1.102 1.101 1.109 1.098 1.095 1.095 1.093 1.092 1.091 1.090 1.090	I.12474 I.12375 I.12277 I.12178 I.12079 I.11081 I.11784 I.1178	1.060 1.059 1.058 1.057 1.056 1.053 1.052 1.051 1.050 1.049 1.045 1.041 1.043 1.042 1.041 1.040 1.040	1.07646 1.07547 1.07448 1.07350 1.07252 1.07153 1.07055 1.06057 1.06563 1.06464 1.06366 1.06267 1.06169 1.06070 1.05072 1.05874 1.05874 1.05874	I.010 I.009 I.008 I.007 I.006 I.005 I.004 I.001 I.000 I.001 I.000 I.098 I.099 I.090	1.02726 1.02726 1.02529 1.02431 1.02332 1.02234 1.02136 1.01034 1.01743 1.01546 1.01546 1.01549 1.01531 1.01553 1.01055 1.00657 1.00858 1.00760 1.00669	.960 .959 .958 .957 .956 .955 .953 .952 .951 .950 .949 .948 .947 .946 .945 .944 .943 .941 .949	.97813 .97715 .97617 .97519 .97421 .97323 .97225 .96930 .96832 .96734 .96636 .96336 .96440 .96344 .96146 .96244 .96146 .96349 .96348 .96348 .96348 .96348	.910 .909 .908 .907 .906 .905 .902 .901 .900 .899 .898 .897 .896 .895 .894 .893 .893 .893 .893 .893	.92911 .92813 .92715 .92617 .92519 .92421 .92324 .92326 .92128 .92030 .91032 .91638 .91540 .91442 .91345 .91247 .91449 .91051 .91051
1.085 1.087 1.086 1.085 1.082 1.082 1.081 1.080 1.070 1.075 1.075 1.075 1.074 1.075 1.070 1.070 1.068 1.067 1.065 1.065 1.065 1.065 1.063 1.062 1.061	1.10404 1.10305 1.10307 1.10108 1.10009 1.09911 1.09517 1.09418 1.09221 1.09221 1.09221 1.09221 1.0823 1.08532 1.08532 1.08533 1.08533 1.08533 1.08533 1.08533 1.08533 1.08534 1.08744 1.0744	1.039 1.038 1.037 1.036 1.035 1.033 1.033 1.032 1.030 1.028 1.027 1.026 1.025 1.025 1.021 1.021 1.020 1.018 1.015 1.015 1.015	1.05578 1.05480 1.05283 1.05283 1.05283 1.05283 1.05283 1.05288 1.04988 1.04988 1.04490 1.04490 1.04490 1.04200 1.04020 1.0402 1.0402 1.03807 1.03512 1.03512 1.03512 1.03316 1.03316 1.03316 1.03217 1.03217 1.03217 1.03221	.988 .987 .987 .986 .985 .984 .983 .982 .981 .980 .978 .977 .976 .975 .974 .973 .969 .968 .969 .968 .969 .969 .969 .969	1,00063 1,00465 1,00269 1,00270 1,00270 1,00270 1,0007	939 938 937 936 935 934 933 933 932 937 929 928 927 926 927 928 929 929 929 910 910 910 911 911	-95/53 -95/53 -95/55 -95/557 -95/561 -95/63 -95/65 -95/667 -94/73	.838 .8887 .8886 .8881 .8882 .8882 .8882 .8880 .879 .876 .875 .876 .875 .876 .875 .876 .875 .876 .866 .866 .866 .866 .866 .866 .866	-90055 -90055 -90757 -90660 -90562 -90464 -90366 -90170 -90273 -89075 -89081 -89075 -89081 -89075 -89081 -89081 -89081 -89090 -88087 -89090 -88097 -88097 -88097 -88097 -88097 -88097 -88097 -88097 -88097 -88097 -88097 -88097 -88809 -88909 -89009 -89000 -89000 -89000 -89000 -89000 -89000 -89000 -89000 -89000 -8

TABLE 4. — FOR FINDING DIAMETERS OF CIRCULAR FORMING Tools for Brown & Sharpe No. 2 Automatic SCREW MACHINE

No. 2		No. 2		No. 2		No. 2		No. 2	
Base Line	Radius	Base Line	Radius	Base Line	Radius	Base Line	Radius	Base Line	Radius
I.479	1.49998	I.430	1.45168	1.380	1.40246	1.330	1.35329	1.280	1.30419
I.478	1.49899	I.429	1.45070	1.379	1.40148	1.329	1.35231	1.279	1.30320
I.477	1.49801	I.428	1.44982	1.378	1.40050	1.328	1.35133	1.278	1.30222
I.476	1.49702	I.427	1.44873	1.377	1.39952	1.327	1.35035	1.277	1.30124
I.475	1.49604	I.426	1.44775	1.376	1.39853	1.326	1.34936	1.276	1.30026
I.475	1.49505	I.425	1.44676	1.375	1.39754	1.325	1.34838	1.275	1.29928
I.473	1.49406	1.424	I.44578	I.374	1.39656	I.324	I.34740	I.274	1.29830
I.472	1.49308	1.423	I.44479	I.373	1.39558	I.323	I.34641	I.273	1.29732
I.471	1.49209	1.422	I.44381	I.372	1.39459	I.322	I.34543	I.272	1.29633
I.470	1.49110	1.421	I.44282	I.371	1.39360	I.321	I.34445	I.271	1.29535
I.469	1.49012	1.420	I.44184	I.370	1.39262	I.320	I.31347	I.270	1.29437
I.468	1.48913	1.419	I.44085	I.369	1.39164	I.319	I.34248	I.269	1.29339
1.467	1.48815	1.418	1.43987	1.368	1.39066	1.318	1.34150	1.268	1.29241
1.466	1.48716	1.417	1.43874	1.367	1.38967	1.317	1.34052	1.267	1.29143
1.465	1.48618	1.416	1.43790	1.366	1.38869	1.316	1.33954	1.266	1.29045
1.464	1.48519	1.415	1.43090	1.365	1.38770	1.315	1.33855	1.265	1.28947
1.463	1.48421	1.414	1.43593	1.364	1.38672	1.314	1.33757	1.264	1.28848
1.462	1.48322	1.413	1.43495	1.363	1.38574	1.313	1.33659	1.263	1.28750
1.461 1.460 1.459 1.458 1.457 1.456	1.48223 1.48125 1.48026 1.47928 1.47829 1.47731	1.411 1.410 1.400 1.408 1.407	1.43396 1.43298 1.43199 1.43100 1.43002 1.42905	1.362 1.361 1.360 1.359 1.358 1.357	1.38475 1.38377 1.38279 1.38181 1.38082 1.37984	1.312 1.311 1.310 1.309 1.308 1.307	1.33560 1.33462 1.33364 1.33266 1.33168 1.33069	1.262 1.261 1.260 1.259 1.258 1.257	1.28652 1.28554 1.28456 1.28358 1.28260 1:28162
I.455	I.47632	1.406	1.42805	1.356	I.37885	1.306	1.32971	1.256	1.28064
I.454	I.47534	1.405	1.42707	1.355	I.37786	1.305	1.32873	1.255	1.27966
I.453	I.47435	1.404	1.42608	1.354	I.37689	1.304	1.32775	1.254	1.27868
I.452	I.47337	1.403	1.42510	1.353	I.37590	1.303	1.32677	1.253	1.27770
I.451	I.47238	1.402	1.42411	1.352	I.37492	1.302	1.32578	1.252	1.27672
I.450	I.47139	1.401	1.42313	1.351	I.37393	1.301	1.32480	1.251	1.27574
1.449 1.448 1.447 1.446 1.445 1.444	1.47041 1.46914 1.46846 1.46745 1.46647 1.46548	1.400 1.399 1.398 1.397 1.396 1.395	1.42215 1.42116 1.42118 1.41919 1.41821 1.41723	1.350 1.349 1.348 1.347 1.346 1.345	1.37295 1.37197 1.37098 1.37000 1.36902 1.36804	1.300 1.299 1.298 1.297 1.296 1.295	1.32382 1.32284 1.32186 1.32087 1.31989	1.250 1.249 1.248 1.247 1.246 1.245	1.27476 1.27377 1.27279 1.27181 1.27083 1.26985
1.443	1.46459	1.394	I.41624	I.344	1.36705	I.294	1.31793	I.244	1.26887
1.442	1.46351	1.393	I.41526	I.343	1.36607	I.293	1.31695	I.243	1.26789
1.441	1.46253	1.392	I.41427	I.342	1.36509	I.292	1.31596	I.242	1.26691
1.440	1.46154	1.391	I.41329	I.341	1.36410	I.291	1.31498	I.241	1.26593
1.439	1.46056	1.390	I.41230	I.340	1.36312	I.290	1.31400	I.240	1.26495
1.438	1.45957	1.389	I.41132	I.330	1.36214	I.289	1.31301	I.239	1.26397
I.437	1.45853	1.388	1.41033	I.338	1.36116	1.288	1.31203	1.238	1.26299
I.436	1.45759	1.387	1.40935	I.337	1.36017	1.287	1.31106	1.237	1.26201
I.435	1.45661	1.386	1.40837	I.336	1.35919	1.286	1.31008	1.236	1.26103
I.434	1.45563	1.385	1.40738	I.335	1.35820	1.285	1.30909	1.235	1.26005
I.433	1.45464	1.384	1.40640	I.334	1.35722	1.284	1.30811	1.234	1.25907
I.432	1.45366	1.383	1.40541	I.333	1.35624	1.283	1.30713	1.233	1.25809
1.431	1.45267	1.382 1.381	I.40443 I.40345	I.332 I.33I	1.35526 1.35428	1.282	1.30615 1.30517	I.232 I.23I	1.25711

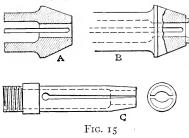
Table 4. — For Finding Diameters of Circular Forming Tools for Brown & Sharpe No. 2 Automatic Screw Machine

N	0. 2	N	0. 2	N	0. 2	N	0. 2	N	No. 2
Base Line	Radius	Base Line	Radius	Base Line	Radius	Base Line	Radius	Base Line	Radius
1.230	1.25515	1.178	1.20424	1.126	1.15342	1.074	1.10271	1.022	1.05213
1.229	1.25417	1.177	1.20326	1.125	1.15244	1.073	1.10174	1.021	1.05116
1.228	1.25319	1.176	1.20228	1.124	1.15147	1.072	1.10076	1.020	1.05010
1.227	1.25221	1.175	1.20130	1.123	1.15049	1.071	1.09979	1.019	1.04920
1.226	1.25123	1.174	1.20032	1.122	1.14951	1.070	1.09882	1.018	1.04825
1.225	1.25025	1.173	1.19934	1.121	1.14854	1.069	1.09784	1.017	1.04728
1.224	1.24927	1.172	1.19837	1.120	1.14756	1.068	1.09687	1.016	1.04631
1.223	1.24829	1.171	1.19739	1.119	1.14659	1.067	1.09590	1.015	1.04533
1.222	1.24731	1.170	1.19641	1.118	1.14561	1.066	1.09492	1.014	1.04436
I.22I	1.24633	1.169	1.19543	1.117	1.14463	1.065	1.09395	I.013 I.012	1.04339
I.220 I.210	1.24535	1.167	1.19446	1.116	1.14366	1.063	1.00200	1.011	1.04242
1.218	1.24437	1.166	1.19340	1.113	1.14200	1.062	1.00103	1.010	1.04048
1.217	1.24241	1.165	1.19152	1.113	1.14073	1.061	1.00006	1.000	1.03051
1.216	1.24143	1.164	1.19054	1.112	1.13976	1.060	1.08008	1.008	1.03854
1.215	1.24045	1.163	1.18957	1.111	1.13878	1.059	1.08811	1.007	1.03757
1.214	1.23947	1.162	1.18859	1.110	1.13780	1.058	1.08714	1.006	1.03660
1.213	1.23849	1.161	1.18761	1.100	1.13683	1.057	1.08616	1.005	1.03563
1.212	1.23752	1.160	1.18663	1.108	1.13586	1.056	1.08519	1.004	1.03466
1.211	1.23654	1.159	1.18566	1.107	1.13488	1.055	1.08422	1.003	1.03369
1.210	1.23556	1.158	1.18468	1.106	1.13390	1.054	1.08324	1.002	1.03273
1.200	1.23458	1.157	1.18370	1.105	I.13293	1.053	1.08227	1.001	1.03175
I.208 I.207	1.23360	1.156	1.18272	1.104	1.13195	1.052	1.08032	.000	1.03078
1.206	1.23164	1.154	1.18077	1.103	1.13000	1.050	1.07935	.998	1.02884
1.205	1.23066	1.153	1.17070	1.101	1.12003	1.049	1.07838	.997	1.02787
1.204	1.22968	1.152	1.17881	1.100	1.12805	1.048	1.07741	.996	1.02600
1.203	1.22870	1.151	1.17784	1.099	1.12708	1.047	1.07643	.995	1.02593
1.202	1.22772	1.150	1.17686	1.098	1.12610	1.046	1.07546	-994	1.02496
1.201	1.22675	1.149	1.17588	1.007	1.12513	1.045	1.07449	993،	1.02399
1.200	1.22577	1.148	1.17491	1.096	1.12415	1.044	1.07352	.992	1.02302
1.199	I.22479 I.2238I	1.147	1.17393	1.095	1.12318	1.043	1.07254	.991	1.02205
1.198	1.22381	1.146	I.17295 I.17197	1.004	I.I2220 I.I2I23	I.042 I.04I	1.07157	.990	1.02108
1.196	1.22185	1.145	1.17100	1.003	1.12123	1.041	1.06963	.988	1.01914
1.105	1.22087	1.143	1.17002	1.001	1.11028	1.030	1.06865	.987	1.01817
1.194	1.21989	1.142	1.16004	1.000	1.11830	1.038	1.06768	.986	1.01720
1.193	1.21891	1.141	1.16807	1.089	1.11733	1.037	1.06671	.985	1.01623
1.192	1.21793	1.140	1.16709	1.088	1.11635	1.036	1.06574	.984	1.01526
1.191	1.21696	1.139	1.16612	1.087	1.11538	1.035	1.06477	.983	1.01429
1.190	1.21598	1.138	1.16514	1.086	1.11440	1.034	1.06379	.982	1.01332
1.189	1.21500	1.137	1.16416	1.085	1.11343	1.033	1.06282	.981	1.01235
1.188 1.187	1.21402	1.136	1.16318	1.084	1.11246	1.032	1.06185	.980	1.01139
1.186	1.21304	1.135 1.134	1.16221	1.083	1.11148	1.031	1.05003	.979 .978	1.01042
1.185	1.21200	1.134	1.16025	1.081	1.11051	1.030	1.05893	.977	1.00848
1.184	1.21011	1.132	1.15928	1.080	1.10856	1.028	1.05796	.976	1.00751
1.183	1.20913	1.131	1.15830	1.079	1.10758	1.027	1.05699	.975	1.00654
1.182	1.20815	1.130	1.15732	1.078	1.10661	1.026	1.05602	.974	1.00557
1.181	1.20717	1.129	1.15635	1.077	1.10563	1.025	1.05505	.973	1.00460
1.180	1.20619	1.128	1.15537	1.076	1.10466	1.024	1.05408	.972	1.00363
1.179	1.20521	1.127	1.15440	1.075	1.10369	1.023	1.05310	.971	1.00267
		1						.970	1.00170
	<u> </u>			1			·	<u> </u>	

HARDENING SPRING COLLETS AND FEED CHUCKS

Before hardening collets it is common practice to open them somewhat to insure their having a given tension after hardening and tempering so that they will open and release the stock the instant they are themselves freed by the chucking mechanism. This opening of the chuck must be carefully attended to or an eccentric and unsatisfactory job will result. Sometimes a simple fixture having a cone pointed spindle is used for this purpose, the collet being held centrally while the cone plunger is forced between the chuck jaws to open them evenly the necessary amount. No matter how much care is taken in this operation, the effect is lost unless the hardening is properly attended to and only grinding will produce a perfectly true collet.

Preventing Distortion. — Some toolmakers take the precaution of leaving a thin fin of metal at the front end of the collet in each saw slot as at A, Fig. 15, in order that when hardened there shall be no chance



of distortion due to unequal springing of the This metal tie or bridge at the ends of the jaws is removed by grinding out with a thin slitting wheel or lap. Another method, shown at B, also leaves a narrow ring at the front end to run on grinder center while collet is ground outside. The ring at the end of the nose

may be ground off leaving the collet ready for use.

Another method of preventing trouble in hardening is to insert a piece of metal, say $\frac{1}{32}$ inch thicker than the width of the saw slot, in the front end of the slots and then wire the nose of the chuck tightly so as to retain the steel pieces during the hardening process. collet must be heated uniformly and dipped so as to insure all three prongs being cooled simultaneously. With the best of care a collet that is hardened but not ground afterward will generally require touching up on the conical portion of one or two of the prongs to insure its running true. It is not difficult, however, to make the collet run true within 0.002 inch by polishing one or two prongs.

In order that the collet may close parallel it must be fairly long and the outside of each prong or jaw may be relieved by filing so as to insure its bearing along the center line on the conical surface. must be carefully tempered at the ends of slots to prevent breaking.

Feed Chucks. — Feed chucks need no such refinement in their They are usually closed after slitting on opposite sides so that after hardening they will maintain a constant grip on the stock sufficient to feed it forward when it is released by the chuck. The idea is indicated at C, Fig. 15. Ordinarily the hole for the stock should be bored a little over size otherwise the corners of the feed chuck jaws when drawn back over the stock will mar the surface.

Table 1. Cutting Speeds and Feeds for Screw Stock

3.2	INCH	CHIP			Ye INCH				\$ INCH (CHIP			J INCH (CHIP	
_	eet	Rev.	Feed	Dia.	Feet	Rev.	Feed	Dia.	_	Rev.	Feed	Die	Foot	Down	Poor
Š	- fore	מסנו	nor	•	Curting	Dor	100	30	Cf.		3	•	10,	TVCV.	T. CCT
j	poor	1.5	100	Ctock	Crood	Vi:	D 21.	5 5	Surrace	Jer.	jer,	10,	Surface	per	per
7		TATILE.	TACK.	STOCK	Specia	TATILL.	Nev.	SLOCK	peade	MIII.	. Kev.	Stock	Speed	Min.	Rev.
	80	2445	.002	-4-	99	916	.0035	mja	75	260	.004	00	S.	0.7.4	3
•	()	900		~	7	9		-	3	,	-	di.	2	+01	5
_	2	1420	500.	90	3	OII	±000*	42	22	420	200.	H	0	101	200
_	20	0901	.00	→ ¢	99	458	200	co -	v	0000	900	1,1	1	1 1	
	_	1	- 1	3 00		20	0,0	4	CC.		3	*	45	13/	500
	- 02	713	500.	4	52	280	000.	_	5.0	161	. 000.	-&-	7.7	III	900
_	200	200	900	,	u	010	100	1	ì	1	. !	9 65	2 !	10	
_	_	2		,	20		7	₹,	200	132	700.	-	5	98	000.
_	90	305	.007	ΙŢ	ν.	168	.007	-\frac{1}{1}	45	LII	100	·	5	1,0	900
_	-	000	000	,,	0 1	1	-0	9 67	2	+ <	/22:	,	+	2	000
_	3	622	000.	1.2	20	127	2000	H	45	93	.007	2.4	40	89	007
_		183	X	6		. 00 1	000	٠,	2 (1	-0	*-) †	3 ,	·
-	,	007	3	*	200	657	3	N	5	2	000.	ć	40) I	002

With high speed steel tools the speeds in these tables may be increased about 30 per cent and feeds 10 to 20 per cent.

CUTTING SPEEDS AND FEEDS FOR BRASS

Table 2.

		Feed	Torr	Rev			000.	1	7	800.	000	000	000	600	600.	
Снір	į	_	-	_	16.0	_	_	-	-	-		-	-	_	_	-
1 INCH		Feet	Surface	Speed	L.	130	150	127	50	1.35	12.0	100	120	00	071	
	-	Dia.	jo	Stock	0	* 1	-	T.	# ,-	Į.	e	4	01	2	4	
	1	reed	per	Řev.	100	900	000.	.007	- 0	000.	000	6	010.	OTO)	
CHIP	1	Kev.	per	Min.	1680	200	7200	840	1	573	456	-	342	207	+0	2000
A INCH	Poot 1	Leel Leel	Surface	Speed	165	10,1	CO.	165	1	150	150	,	135	135		00 1
	1	Dia.	o.	Stock	co)ot	o ~ 4		গৰ	,	-	I	-	\$7 T	e+3	*	•
	Food	T. CC.	ber	Rev.	.004	. 1	C \	.0005	1	15/00:	2800.	,	010.	.012	-	010
т Спр	Roy		ioi;	Min.	2748	1833	-000	1374	×	040	630		204	381		100
Te INCH	Foot	10.7	Sarrace	Speed	180	180	2	180	16.	S	165	, 6, 4	105	150)	-
				Stock		mi	×,-	(C1	(c)	#	ı			15	07	*
	Food	7.7	per	Kev.	.001	100	1	.005	900	300	200.	0,0	5	·OII.	-	0
н Снір	Rov	-	jer Z	Min.	5500	2668	9	2748	1822	1000	1374	1	915	087		
32 INCH	Popt	7,7	Surface	Speed	180	180		180	200	201	180	00.1	001	180	00-	000
- 1	_		5,0	.1		∞¦,		4	mi	20 -	€ €3	63	4	н		-

SPEEDS AND FEEDS FOR SCREW MACHINE WORK

THE accompanying tables of speeds and feeds for different types of tools used on materials commonly worked in the automatic screw machine have been compiled from data accumulated and thoroughly tested during extended experience in this class of work.

It is, of course, impossible, where a series of tools is used on an automatic machine, to select speeds theoretically correct for every tool carried by the turret and cross slide. A compromise is necessary and therefore speeds are selected which will fall within the range

suitable for the different tools.

Speeds and Feeds for Turning

Tables 1 and 2, page 261, cover turning speeds and feeds for bright-drawn stock (screw stock) and brass, with various depths of chip (that is, stock removed on a side) from $\frac{1}{32}$ up to $\frac{1}{4}$ inch. These feeds and speeds and depths of cut are figured more especially for such tools

TABLE 3. S	SPEEDS A	ND]	FEEDS	FOR	FINISH	Box	Tool
------------	----------	------	-------	-----	--------	-----	------

노를 것 목		Screw Sto	СК		Brass Rod	•	nt ole ove de
Finished Diameter of Work	Feet Surface Speed	Rev. per Min.	Feed per Rev.	Feet Surface Speed	Rev. per Min.	Feed per Rev.	Amount advisable to remove on a side
18 3 16 14 12 23 4 1 12 2	80 70 65 60 60 60 55	2445 1426 993 458 305 229 140	.0045 .0055 .0075 .011 .012 .012	180 180 180 180 180 175 170	5500 3668 2750 1375 917 668 433 325	.0045 .0055 .0075 .011 .012 .012 .014	.0025 .0025 .0045 .006 .006 .0065 .007

With high speed steel tools the above speeds may be increased about 30 per cent and feeds 10 to 20 per cent.

as roughing boxes where the cut, though frequently heavy, is taken by a single cutting tool. For a $\frac{1}{16}$ -inch chip the feeds for various diameters of stock are practically midway between those tabulated for $\frac{1}{8}$ -and $\frac{1}{4}$ -inch chips. The feed per revolution for $\frac{3}{8}$ chip taken on diameters $\frac{1}{4}$ inch and larger is the same as given for $\frac{1}{4}$ inch chip, the speed also being the same for corresponding diameters. Where hollow mills are used on steel and the work is divided among three or more cutting edges the feed per revolution for a given depth of chip is about 25 per cent coarser than given for box tools; with both classes of tools the feeds are, of course increased as the diameter of

the stock increases, the peripheral speeds being reduced as the feeds grow coarser. The speeds and feeds for finishing box tools as used on screw stock and brass are given in Table 3, the last column indicating the amount of stock which, generally speaking, it is advisable to remove in order to produce a good surface.

Forming-tool Speeds and Feeds

Speeds and feeds for forming tools are given in Tables 4 and 5. It will be seen that after a work diameter of about $\frac{1}{4}$ inch has been reached, a tool about $\frac{1}{8}$ -inch wide is adapted to take the coarsest feed, tools from this width up to approximately $\frac{8}{16}$ (such as commonly employed for cutting-off purposes) admitting of heavier crowding as a rule than either the narrower or wider tools.

Table 4. Speeds for Forming

Work	Screw	Stock	Brass	Rod	Work	SCREW	Stock	BRASS	s Rod
Dia. of 1	Feet Surface Speed	Rev. per Min.	Feet Surface Speed	Rev. per Min.	Dia. of V	Feet Surface Speed	Rev. per Min.	Feet Surface Speed	Rev. per Min.
183 16 1438 12	75 75 70 65 65	2292 1528 1069 662 497	200 200 185 185 185	6112 4074 2827 1885 1414	58 34 I I 1½ 2	60 60 60 60 50	360 305 229 153 96	175. 175 175 170 170	1050 882 667 432 324

With high speed steel tools the above speeds may be increased about 30 per cent.

Table 5. Feeds for Forming Tools

Width	Smallest Diameter of Form													
of Form	16	1 8	3 16	14	3 8	1/2	3 4	1 1/2						
1 166 14 3 5 12 3 4 I	.0007	.0008	.001 .001 .000 .0009	.0012 .0012 .001 .001 .0009	0012 .0015 .0015 .001 .001 .0009 .0008	.0012 .0020 .0015 .0012 .001 .001 .0009	.0012 .0025 .0018 .0015 .0015 .0011 .0009 .0007	.0012 .0025 .0018 .0015 .0015 .0012 .0012						

With cutting-off tools of high speed steel the above feeds may be increased to to per cent.

Drilling Speeds and Feeds

Drilling speeds and feeds are given in Table 6. While these speeds are based on much higher peripheral velocities than drill-makers as a rule recommend for general purposes, it should be noted that conditions for drilling in the automatic, on the ordinary run of work, are usually ideal so far as lubrication, steadiness of feed, etc., are concerned, and it is possible where the holes drilled are comparatively shallow and the drill has ample opportunity for cooling during operation of the other tools, to maintain speeds that would be considered too high to be attempted in general shop practice.

TABLE 6. DRILLING FEEDS AND SPEEDS

	Screw	Stock	Brass	Rod	_	Screw	Stock	Brass	Rod
Dia. of Drill	Feed per Rev. of Drill	R.P.M. at 60 Ft. Periph- eral Speed	Feed per Rev.	R.P.M. at 175 Ft. Periph- eral Speed	Dia. of Drill	Feed per Rev.	R.P.M. at 55 Ft. Periph- eral Speed	Feed per Rev.	R.P.M at 165 Ft. Periph- eral Speed
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$.0013 .0016 .0018 .0025 .003 .004 .0045 .0045 .0045 .0045	3667 2933 2445 1833 1421 1222 1048 916 815 733 611 524	.0017 .002 .0023 .0033 .0039 .0052 .0058 .0058 .0058 .0061	10696 8555 7130 5348 4144 3565 3050 2674 2377 2139 1783 1528	$\begin{array}{c} \frac{1}{2} \\ \frac{9}{16} \\ 5 \\ 5 \\ 8 \\ \frac{1}{16} \\ \frac{3}{4} \\ \frac{4}{78} \\ 8 \\ \\ I \\ I \\ \frac{1}{2} \\ \frac{3}{4} \\ 2 \\ \end{array}$.005 .0057 .0059 .006 .0065 .0075 .0085 .0095 .011	420 373 336 305 280 240 50 Ft. 191 152 127 109 96	.0065 .0074 .0077 .0078 .0084 .0097 .0110 .0123 .0143 .0169 .0182	1260 1120 1008 917 84 702 150 Ft. 573 458 382 327 294

With high speed drills the above speeds may be increased about 30 per cent.

Speeds and Feeds for Reaming

Table 7 is made up of speed and feed data for reamers. In this table the feed for different classes of material has been considered as constant for any given diameter of reamer, although it is probable that with certain materials, especially on brass alloys, etc. the feed per revolution might be increased somewhat, to advantage, over the rates given. These feeds have been tabulated, however, as representing highly satisfactory practice in reaming the materials listed.

TABLE 7. REAMING FEEDS AND SPEEDS

mer		Amount	REV. PE	R MIN.	mer		Amount	REV. PI	ER MIN.
Dia. of Reamer	Feed per Rev.	to Remove on Dia.	Screw Stock at 40 Ft.	Brass Rod at 130 Ft.	Dia. of Reamer	Feed per Rev.	to Remove on Dia.	Screw Stock at 40 Ft.	Brass Rod at 13° Ft.
18	.005	.0045	1222	3972	$I_{\frac{1}{4}}$.018	.010	I 22	397
1836	.006	.0045	815	2648	$1\frac{1}{2}$ $1\frac{3}{4}$.020	.010	102	331
14	.007	.006	611	1986	1 3/4	.022	.010	87	284
38	.0085	.006	407	1324	2	.024	.013	76	248
$\frac{1}{2}$.0105	.008	306	993	21/4	.026	.013	68	220
5	.012	.008	245	795	$2\frac{1}{2}$ $2\frac{3}{4}$.028	.013	61	199
34	.014	.008	204	662	$2\frac{3}{4}$.030	.013	56	181
1	.016	.010	153	497	3	.032	.013	51	165

With high speed reamers the above speeds may be increased about 20 per cent

Threading, Counterboring, Etc.

Table 8 explains itself and, while giving speeds for threading work with dies, should be of equal value in establishing speeds for tapping.

Table 8. Speeds for Dies. Standard Threads
WITH HIGH-SPEED STEEL DIES

hread	Screw	Ѕтоск	BRASS	Rod	of Thread	SCREW	Stock	Brass	Rod
Dia. of Thread	Feet Surface Speed	Rev. per Min.	Feet Surface Speed	Rev. per Min.	Dia. of Tl	Feet Surface Speed	Rev. per Min.	Feet Surface Speed	Rev. per Min.
18 14 38 12 58	40 40 35 35 35	1222 611 356 267 210	135 125 120 120	4126 1909 1222 917 715	$ \begin{array}{c} \frac{3}{4} \\ I \\ I \frac{1}{4} \\ I \frac{1}{2} \\ 2 \end{array} $	35 30 30 30 25	178 115 92 76 48	115 110 100 90 85	586 420 306 229 162

For carbon steel dies run at 50 to 75 % of above speeds.

For feeds for counterbores from \(\frac{3}{8}\) inch to 2 inches diameter, Tables I and 2 for turning may be followed where the counterbores cut to a depth from one half to three quarters their diameter. Where cutting deeper than about one diameter, the feeds should be decreased; in such depths it is well to withdraw the counterbore during the cutting operation to free it from chips.

PUNCH PRESS TOOLS

METHOD OF FINDING THE DIAMETERS OF SHELL BLANKS

This method for the finding of diameters of shell blanks, applies

also to some other shapes which frequently occur in practice.

The method is based upon the surface of the shell in comparison with the area of the blank and should therefore be used only when light material is to be considered. In case of the flanged shapes the width of the flange should be small in proportion to the diameter.

CYLINDRICAL SHELL

Fig. 1 shows a cylindrical shell of the diameter d and the depth h. To find the diameter of the blank, lay down the diameter d of the shell twice on a horizontal line, Fig. 2, add to this a distance equal to four times the depth h of the shell and describe a semicircle of which the total distance is the diameter. The vertical line D from the intersecting point with the circle to the horizontal line gives the desired blank diameter. Line D is to be drawn at a distance d from the end of the horizontal.

FLANGED SHELLS

If the shell has a flange as in Fig. 3, add four times the width of this flange to the horizontal line and proceed as above; see Fig. 4.

In the case of a hemisphere, Fig. 5, lay down the diameter three times on the horizontal line and draw the vertical line at the dis-

tance d from the end, as in Fig. 6.

If the hemisphere has a flange as in Fig. 7, add a distance equal to twice the width of the flange to the horizontal line, as in Fig. 8. In any case, the length of the vertical line D gives the desired diameter of blank.

TAPER SHELLS

If a shell with tapering sides, Fig. 9, has to be drawn, multiply first the bottom diameter by itself and divide the product by the sum of the two diameters d_1 and d in order to obtain the length x. Otherwise proceed as shown in Fig. 10.

FLANGED TAPER SHELLS

If the taper shell has a flange of the width a, Fig. 11, add to the base line of the diagram twice this width, as shown in Fig. 12.

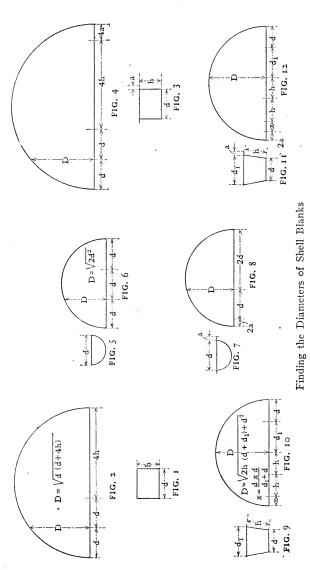


TABLE OF DIAMETERS OF SHELL BLANKS

THE table shows the diameters of blanks for shells $\frac{1}{4} \times \frac{1}{4}$ inch to 6×6 inches inclusive, by $\frac{1}{4}$ inches. The figures were obtained by the formula given on page 267:

$$D = \sqrt{d(d + 4h)}$$

where,

d = Diameter of finished shell.h = Hight of finished shell.

They were also checked by figuring on the area of the metal.

If it is desired to punch the metal in one or more operations, get the mean hight of the shell by the following formula;

 $m = \frac{h t}{T}$

where

m = Mean hight of finished shell.

h = Hight of finished shell.

t = Thickness of finished shell.

T =Thickness of metal before drawing.

Suppose for example, a shell 2 inches diameter by 6 inches high; thickness of metal before drawing, 0.040 inch; finish thickness of shell, 0.020 inch. Then

$$m = \frac{h t}{T} = \frac{6 \times 0.020}{0.040} = 3 \text{ inches.}$$

By using this hight, from the table we find a shell 2 inches diameter by 3 inches high requires a blank 5.29 inches diameter.

When the shell has rounded corners at the bottom, subtract the radius of the corner from the figures given in the table. Thus, in



FIG. 13

the last example, suppose the shell to have a radius of $\frac{1}{8}$ inch on the corner; $\hat{5}.29 - \hat{0}.125 = 5.165$ inches, the required diameter of the blanks.

When a shell has a cross-section similar to the ones shown in Fig. 13, the required blank diameter may be calculated by the following formula:

 $d = 1.1284 \sqrt{\frac{W}{m}}$

where

d = Diameter of blank in inches;

W =Weight of shell;

w = Weight of one cubic inch of the metal;

t = Thickness of shell.

Diameter of Blanks for Shells, $\frac{1}{4} \times \frac{1}{4}$ Inch to 6×6 Inches

Shell	neter		,				Hight	of She	ell				
Shell	Dian	1"	1/2"	3"	ı"	11"	I ½"	I 3"	2"	21"	21/2	2 3/1	3"
1, 1, 1, 1, 1, 1, 1, 1, 1, 1, 1, 1, 1, 1		0.56 0.87 1.41 1.68 1.94 2.190 2.45 2.45 2.45 3.21 3.46 3.71 4.72 4.78 5.73 5.98 6.23 6.48	1.12 1.44 1.73 2.01 2.29 2.56 2.83 3.09 3.36 3.61 3.87 4.13 4.49 5.15 5.41 5.66	0.90 1.32 1.68 2.00 2.30 2.60 2.88 3.16 3.44 3.71 3.98 4.24 4.51 4.77 5.03 5.55 5.81 6.58 7.09 7.35	E 1.030 1.89 2.24 2.556 2.87 3.17 3.46 3.75 4.03 4.31 4.58 4.85 5.12 5.39 6.19 6.71 6.97 7.75	1.14 1.66 2.08 2.45 2.79 3.12 3.44 4.04 4.33 4.02 5.18 5.45 5.73 6.27 6.80 7.07 7.33 7.60 7.86 8.12	1.25 1.80 2.25 2.65 3.36 3.68 4.61 4.61 4.61 5.20 5.48 5.77 6.05 6.32 6.60 6.87 7.15 7.42 7.68 8.22 8.49	1.35 1.94 2.41 2.83 3.21 3.57 3.91 4.24 4.87 5.18 5.48 5.47 6.63 6.63 6.63 6.63 7.19 7.47 7.75 8.29 8.56 8.83	1.44 2.06 2.56 3.00 3.40 4.13 4.47 6.04 6.34 6.64 6.64 6.93 7.22 7.58 8.06 8.34 8.62 8.89 9.17	1.52 2.18 2.70 3.16 3.97 4.34 4.69 5.68 5.68 6.00 6.31 6.61 7.21 7.50 7.80 8.37 8.08 8.37 8.09 9.21 9.49	1.60 2.29 2.84 3.32 3.75 4.53 4.90 5.25 6.56 6.56 6.56 6.56 7.18 8.08 8.37 8.08 8.95 9.23 9.23 9.980	1.68 2.40 2.97 3.46 3.91 4.33 4.72 5.46 5.81 6.48 6.80 7.12 7.44 7.75 8.35 8.05 8.35 8.94 9.24 9.24 9.81	1.75 2.50 3.09 3.61 4.07 4.50 4.50 5.66 6.02 6.37 6.71 7.04 7.36 7.69 8.80 8.31 8.62 9.22 9.52 9.52 9.52 9.52 9.53 10.10
Shell Diameter	3	<u>"</u>	3½"	33"		I 41/1	light o	of Shel	1 5"	51"	517	53"	6"
1 1 1 1 1 2 2 2 3 3 3 3 4 4 4 4 5 5 5 5 5 6	11 22 33 33 34 44 44 55 55 56 66 66 77 77 77 88 88 89 99 100 100 100 100 100 100 100 100 100	.82 .60 .21 .74 .22 .66 .08 .86 .22 .58 .93 .60 .92 .25 .50 .93 .87 .91 .93 .93 .93 .93 .93 .93 .93 .93 .93 .93	1.89 2.69 3.33 3.87 4.82 5.66 6.05 6.42 7.49 7.49 8.49 8.49 9.44 9.44 10.36 10.36 10.36 10.95	1.95 2.78 3.44 4.00 4.51 4.98 5.43 6.23 6.61 6.99 7.35 7.70 8.38 8.72 9.04 9.37 9.69 10.00 10.31 10.62 11.23	2.01 2.87 3.54 4.12 4.64 5.12 5.58 6.00 6.41 6.88 7.55 7.91 8.94 9.28 9.60 9.93 10.25 10.87 11.18	2.08 2.96 3.65 4.24 4.27 5.73 6.16 6.58 6.98 7.77 7.75 8.11 8.82 9.17 9.84 10.16 10.49 10.81 11.12	2.14 3.04 3.75 4.36 4.91 5.41 5.88 6.32 6.75 7.16 7.55 7.94 8.31 8.31 9.38 9.38 9.38 9.39 10.40 10.40 11.05 11.37 11.65 11.37	2.19 3.12 3.85 4.47 5.03 5.55 6.03 6.48 6.91 7.33 7.73 8.12 8.50 8.87 9.24 9.59 9.94 10.28 10.62 11.62 11.93	2.25 3.21 3.95 4.58 5.168 6.17 6.63 7.07 7.50 7.91 8.31 8.69 9.07 9.44 9.80 10.15 10.84 11.18 11.84 12.17	2.30 3.29 4.04 4.69 5.27 5.81 6.78 7.68 8.08 8.49 8.88 9.26 9.63 10.00 10.36 10.71 11.06 11.74 12.07	2.333.41.4.88.5.33.5.99.6.44.6.99.88.22.88.60.99.44.99.88.10.22.10.50.11.22.11.60.11.20.11	5 2.41 6 3.44 8 4.22 9 5.50 1 6.66 6 5.87 7 7.54 7 7.54 9 9.24 9 9.24 9 9.24 9 9.24 9 10.39 9 10.39 9 11.12 1 11.83 1 12.85 1 12.85 1 12.85 1 12.85	2.46 3 50 4.31 5.00 6.18 6.721 7.21 7.69 8.58 9.04 9.04 10.20 10.58 10.96 11.32 11.69 12.04 12.39 12.74 13.08

PUNCH AND DIE ALLOWANCE FOR ACCURATE WORK

In the blanking, perforating and forming of flat stock in the power press for parts of adding machines, typewriters, etc., it is generally desired to make two different kinds of cuts with the dies used. First. to leave the outside of the blank of a semi-smooth finish, with sharp corners, free from burrs, and with the least amount of rounding on the cutting side. Second, to leave the holes and slots that are per-

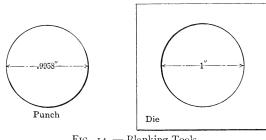


Fig. 14. — Blanking Tools

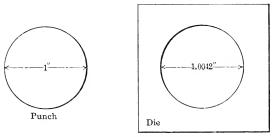


Fig. 15. — Perforating Tools

forated in the parts as smooth and straight as possible, and true to size. The table given is the result of considerable experimenting on this class of work, and has stood the test of years of use since it

was compiled.

The die always governs the size of the work passing through it. The punch governs the size of the work that it passes through. In blanking work the die is made to the size of the work wanted and the punch smaller. In perforating work the punch is made to the size of the work wanted and the die larger than the punch. clearance between the die and punch governs the results obtained.

Figs. 14 and 15 show the application of the table in determining the clearance for blanking or perforating hard rolled steel .060 inch thick. The clearance given in the table for this thickness of metal is .0042, and Fig. 14 shows that for blanking to exactly 1 inch diameter this amount is deducted from the diameter of the punch, while for perforating the same amount is added, as in Fig. 15, to the diameter of the die. For a sliding fit make punch and die .00025 to .0005 inch larger; and for a driving fit make punch and die .0005 to .0015 inch smaller.

Table of Allowances for Punch and Die for Different Thickness and Materials

Thickness of Stock Inch	Clearance for Brass and Soft Steel Inch	Clearance for Medi- um Rolled Steel Inch	Clearance for Hard Rolled Steel Inch
.010	,0005	.0006	.0007
.020	.001	.0012	.0014
.030	.0015	.0018	.0021
.040	.002	.0024	.0028
.050	.0025	.003	.0035
.060	.003	.0036	.0042
.070	.0035	.0042	.0049
.080	.004	.0048	.0056
.090	.0045	.0054	.0063
.100	.005	.006	.007
.110	.0055	.0066	.0077
.120	.006	.0072	.0084
.130	.0065	.0078	.0091
.140	.007	.0084	.0098
.150	.0075	.009	.0105
.160	.008	.0096	.0112
.170	.0085	.0102	.0119
.180	.009	.0108	.0126
.190	.0095	.0114	.0133
.200	.010	.012	014

CLEARANCE FOR PUNCHES AND DIES FOR BOILER WORK

The practice of the Baldwin Locomotive Works on sizes up to $\mathbf{1}_{4}^{1}$ inches is to make the punch $\frac{1}{64}$ inch below nominal size and the die $\frac{1}{64}$ inch above size, which gives $\frac{1}{32}$ inch clearance. Above $\mathbf{1}_{4}^{1}$ inches the punches are made to nominal size and the dies $\frac{1}{32}$ inch large, which allows the same clearance as before. The taper on dies below $\mathbf{1}_{4}^{1}$ inches is 1 inch in 12; on sizes above $\mathbf{1}_{4}^{1}$ inches it is half this or $\frac{1}{2}$ inch in 12 inches.

LUBRICANT FOR PRESS TOOLS

Although there are some shops in which no lubricant is used when working sheet metal, and where good results are obtained, still it is best to use a lubricant on all classes of sheet-metal work.

For all cutting dies on brass and steel a heavy animal oil is best.

Pure lard oil is very satisfactory, although expensive.

When punching copper, or German silver, a thin coating of lard oil or sperm oil should be spread over the sheets or strips before punching. A good way to do this evenly is to coat one sheet thickly and then feed it through a pair of rolls, after which a number of other sheets may be run through the rolls and thus coated evenly. drawn work this method of coating the sheets from which the shells are to be drawn will be found to be the best, as the coating of oil on the stock will be very thin and it will not be found necessary to clean the shells afterward, the oil having disappeared during the blanking and drawing process. When oil is applied with a pad or brush the coating will be so thick that it will be necessary to clean the article produced.

Drawing Steel Shells

In drawing steel shells a mixture of equal parts of oil and black lead is very useful, and while it may be used warm it does not affect the work as much as the speed of the drawing press does; the thicker the stock the slower must be the speed of the punch. A heavy grease with a small proportion of white lead mixed in with it is also recommended for this purpose.

If the drawing die is very smooth and hard at the corner of the "draw," or edge of the die, the liability of clogging will be reduced to a minimum. Often it will help to give to the die a lateral polish by taking a strip of emery cloth and changing the grain of the polish from circular to the same direction as the drawing.

Lubricants for Brass

For drawing brass or copper a clean soap water is considered most satisfactory. One of the largest brass firms in this country uses a preparation made by putting 15 pounds of Fuller's soap in a barrel of hot water, and boiling until all the lumps are dissolved. This is used as hot as possible. If the work is allowed to lie in the water until a slime has formed on the shell it will draw all the better. A soap that is strong in resin or potash will not give good results.

In drawing zinc the water should be hot, or the percentage of broken

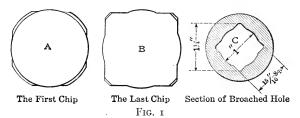
shells will be large.

Aluminum is an easy metal to draw, but it hardens up very quickly. For lubricants lard oil, melted Russian tallow and vaseline are all good. The lubricant should be applied to both sides of the metal.

BROACHES AND BROACHING

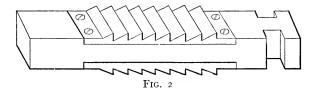
BROACHING is being used more and more to finish holes and even for slots and the outside of pieces of work. In most cases it is used to change a round hole to a square or other shape, such as the four or ten key ways used in automobile transmission.

The chip cut by each tooth varies from 0.001 to 0.007 inch, according to the material being cut and the accuracy required. The teeth are usually undercut from 6 to 10 degrees to give a curl to the chip, while the top clearance is about 30 degrees. Some English practice



undercuts 25 degrees, having top nearly flat. The distance between teeth varies according to the length of the hole being broached, the spacing being larger for long holes so as not to have too many teeth engaged at once, three being a good number. Spacing varies with length of hole.

In broaching square holes from the round, or in other cases where there is a decided change of shape, the first teeth take the widest cut as at A, Fig. 1. This evens up the work of the different teeth

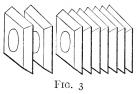


as to the length of surface cut as the hole approaches a square as seen at B.

The blank for the cutting part of each broach is first turned taper by an amount equal to the total cut of the teeth. The tooth spaces are then turned $\frac{1}{2}$ inch apart and about $\frac{3}{2}$ inch deep, this depending on the diameter of the broach, as it must not be unduly weakened; it is then milled $\frac{1}{1}$ inch square as shown. The longer the hole the more chip room must be provided.

Where the hole or other surface to be broached is short, the teeth are often cut on an angle to give a shearing cut. This is also done to prevent chatter at times, another remedy being to space the teeth unevenly as with reamers.

The solid broach is used more than any other. But as tool steel is apt to spring in hardening, and to break out teeth at times, some



use built-up or sectional broaches, especially on large work where the solid broach costs heavily. Some use low carbon steel, case hardened. sectional broaches are made in a variety of ways, Figs. 2 to 4 showing a few examples. In Fig. 2 sections are set in on the side, while in Figs. 3 and 4 the sections are practically disks held on a central arbor. In some cases several

teeth are made on one section. Fig. 4 is made in the same way for broaching internal gears having 66 teeth, 20 diametral pitch and \frac{1}{2} Each tooth cuts 0.006, the last three teeth being straight to insure the size being accurate.



FIG. 4

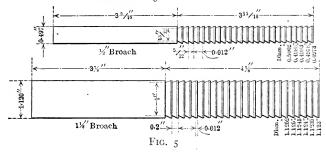
BROACHING ROUND HOLES

Round holes have been broached instead of reamed in some places for many years and the practice is growing. It was formerly confined to soft metal, such as shaft bearings, but is now being made to cover all the metals, in some few cases broaching from a cored hole. For small work a small arbor press with a sort of sub-press can be used to advantage. For larger work the arbor press operated by power is very good and of course the regular broaching machine can be used in any case.

Two broaches used in one shop are shown in Fig. 5, other sizes can be made in proportion. These were used in a hand arbor press. The first 5 or 6 teeth do most of the cutting as these broaches only finish the holes instead of reaming. In some cases with broaches for soft metal bearings and even in cast iron, the large end is left plain and a trifling amount larger than the last tooth. It then acts as a burnisher and compresses the metal. This requires a large amount of power.

In broaching round holes in cast iron, the broach was made from 0.0002 to 0.0003 inch larger than the nominal size and the land was 0.012 as shown. The holes were drilled close to size so as to leave very little work for the broach. In this case about 0.002 inch was left for broaching.

The comparison between broaching and reaming in this case is interesting. The reamers would wear appreciably below size in 25 holes while one broach finished 5000 holes to size.



TO SAVE TIME IN BROACHING OUT SQUARE HOLES

The fit of the gears on a square shaft depends almost entirely on the flat surfaces at or near the corners. With this in mind, it is an economy to bore or drill the round hole in the gear slightly larger than the diameter across the flats of the squared shaft, as shown in Fig. 6.

Taking a r_4^3 -inch square shaft and borning the hole $\frac{1}{16}$ inches in diameter, we see in the illustration exactly what this would mean. The amount of metal to be cut out would be materially reduced, the portion A to B not being touched by the broach in any way. Yet the remaining surface in

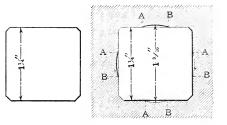
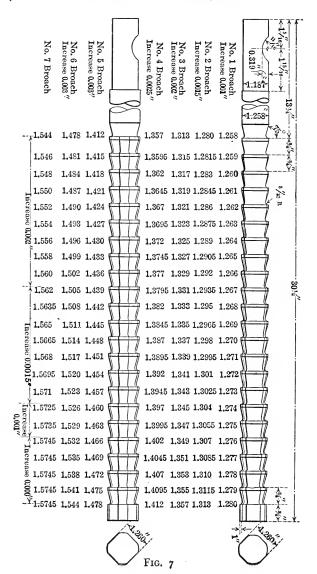


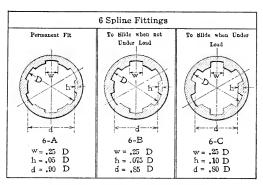
Fig. 6

the corners would be ample to carry all the load of the gears at work, and the clearance A to B would allow the best of lubrication.

The center relief, as shown, gives considerable added chip space as well as reduces the amount of chip, thus allowing a heavier chip per tooth. This may either reduce the length of the broach or allow a longer hole (such as two gears at once) to be broached with the same length of broach.

The set of 7 broaches shown in Fig. 7 show the practice of the Brown & Sharpe Mfg. Co. in making automobile transmission gears. The gears are of a tough alloy steel making the 7 necessary to secure an accurate hole of r_4^1 inches across the flats. Each broach is $30\frac{1}{2}$ inches long, the cutting portion being only $17\frac{1}{2}$ inches. The method of holding the shank can be readily seen.



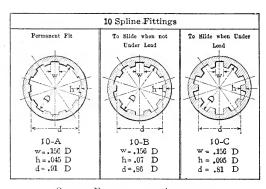


6-Spline Fittings for Automobiles

From sixth Report of Broaches Division S.A.E. Accepted at Meeting of Society, January, 1914

Nominal Diam.	D	d	w	T	D	<i>d</i>	w	T	D	d	w	Т
34	.750 .749	.675 .674	.188 .187	80	.750 .749	.638 .637	.187	117	.750 -749	.600 •599	.188 .187	152
78	.875 .874	.788 .787	.219	109	.875 .874	·744	.219	159	.875 .874	.700 .699	.219	207
I	000.1	.900	.250	143	000.1	.850 .849	.250	208	1.000	.800	.250	270
$I\frac{1}{8}$	1.125	1.013	.281	180	1.125	.956 1.955	.281	263	1.125 1.124	.900 .899	.281	342
I 1	I.250 I.240	I.125 I.124	.313	223	I.250 I.240	1:063 1.062	.313	325	1.250	1.000	.313	421
I 3	1.375 1.374	1.238	-344 -343	269	1.375 1.374	1.169 1.168	·344 ·343	393	1.375	1.100	-344 -343	510
$I^{\frac{1}{2}}$	1.500 1.400	1.350 1.349	·375	321	I.500 I.499	1.275 1.274	·375	468	1.500	1.200	·375	608
I 5	1.625 1.624	1.463 1.462	.406	376	~ 6 1	1.381	.406	550	1.625	1.300	.406	713
1 3	I.750 I.740	1.575 1.574	.438	436	1.750	1.488	.438 .437	637	1.750	1.400	.438	827
2	2.000 1.998	1.800	.500	570	2.000 1.998	1.700	.500	833	2.000 1.998	1.600	.500	1080
2 4	2.250	2.025	.563 .561	721	2.250	1.013	.563 .561	1052	2.250	1.800	.563	1367
$2\frac{1}{2}$	2.500 2.498	2.250	.625	891	2.500	2.125	.625	1300	2.500	2.000	.625	1688
3	3.000	2.700 2.698	.750 .748	1283	3.000	2.550	.750	1873	3.000	2.400	.750	2450
					1	34-	, 4-		- 995	590	.,40	

 $T=1000\times 6$ (No. of Splines) \times Mean Radius \times $h\times 1$ = inch-pounds torque capacity per inch bearing length at 1000 lbs. pressure per square inch on sides of splines. No allowance is made for radii on corners nor for clearances.



10-Spline Fittings for Automobiles

From Sixth Report of Broaches Division S.A.E. Accepted at Meeting of Society, January, 1914

Nominal Diam.	D	d	w	Т	D	·d	w	T	D	<i>d</i> .	าย	T
Nor Dia												
34	.750 .749	.68 ₃	.117 .116	120	.750 .749	.645 .644	.117 .116	183	.750 .749	.608 .607	.117 .116	241
7 8	.875	.796 .795	.137 .136	165	.875 .874	·753	.137 .136	248	.875	.709 .708	.137	329
I	1.000	.910	.156	215	.000	.860 .859	.156	326	.000	.810	.156	430
$I\frac{1}{8}$	I.I25 I.I24	I.024 I.023	.176	27 I	I.I25 I.I24	.968 .967	.176 .175	412	I.I25 I.I24	.911	.176	545
$I^{\frac{1}{4}}$	I.250 I.240	1.138	.195	336	I.250 I.249	1.075 1.07.1	.195	508	I.250 I.240	1.013	.195	672
I 3/8	I.375 I.374	1.251	.215	406	I.375 I.374	1.183	.215	614	I.375 I.374	1.114	.215	813
$I^{\frac{1}{2}}$	I.500 I.400	1.365 1.364	.234	483	1.500 1.400	1.290 1.289	.234	732	1.500	1.215	.234	967
I 5	1.625 1.624	1.479 1.478	.254	566	1.625 1.624	1.398 1.397	.254	860	I.625 I.624	1.316	.254	1135
I 3	I.750 I.740	I.593 I.592	.273	658	I.750 I.740	1.505 1.504	.273	997	1.750 1.749	1.418	.273	1316
2	2.000 1.998	1.820	.312	860	2.000 1.998	I.720 I.718	.312	1302	2.000 1.998	1.620	.312	1720
$2\frac{1}{4}$	2.250	2.048	.35I -349	1088	2.250	1.935 1.933	.351	1647	2.250	1.823	.351 .349	2176
$2\frac{1}{2}$	2.500	2.275 2.273	.390	1343	2.500 2.498	2.150 2.148	.390	2034	2.500	2.025	.390	2688
3	3.000 2.998	2.730 2.728	.468 .466	1934	3.000	2.580 2.578	.468 .466	2929	3.000 2.998	2.430 2.428	.468 .466	386 9

 $T=1000 \times 10$ (No. of Splines) \times Mean Radius $\times h \times 1=$ inch-pounds torque capacity per inch bearing length at 1000 lbs. pressure per square inch on sides of Splines. No allowance is made for radii on corners nor for clearances.

BOLTS, NUTS AND SCREWS

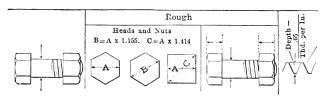
U. S. STANDARD BOLTS AND NUTS

THE U. S. Standard for bolts, nuts, etc., called also Sellers' Standard, Franklin Institute Standard, and American Standard, was recommended in 1864 by the Franklin Institute for general adoption by engineers. (See Note).

STRENGTH OF U. S. STANDARD BOLTS FROM 1/4 TO 3" DIAMETER

Ве	olt	Ar	eas		sile Stre	ngth		Shearing	Strengtl	1
Bolt	No. of Threads per Inch		Thread	. per	ber .	. per	Full	Bolt	Bottom	of Thread
of E	read			lbs.	lbs.	lbs.	ln.	Ps.	lbs. In.	bs.
ter	f _	Bolt	Bottom of	At 10,000 Sq. In.	At 12,500 Sq. In.	At 17,500 Sq. In.		At 10,000 lbs. per Sq. In.		At 10,000 lbs. per Sq. In.
ıme	och nch	B B	tton	Sq.	L 12,	sq.	t 7,5 per ?	t ro,c	e 7.	tro,o per S
Ü	No.	Full	Bot	₹".	At 18	\At	~		At D	At
SLAW OF HOW DE LOW DE LAW DE L	20	.049				470	380	490	200	270
16	18	.077	.045		570	790	580	770		450
8	16	.110	.068		1	1,190	830	1,100		680
16	14	.150						1,500	700	
2	13	.196	.126					1,960	940	
1 6	12	.248		1,620						1,620
8	II	.307	.202	2,020					1,510	2,020
4	10	.442		3,020		5,290			2,270	
	8	.601	.419		5,240	7,340	4,510	6,010	3,150	4,190
I.	1	.785	.551	5,510	6,890	9,640	5,890			
I &	7	-994	.693	6,930	8,660	12,130	7,450			
14	7	1.227	.890	8,890	11,120	15,570	9,200	12,270	6,670	
18	6	1.485	1.054	10,540	13,180	18,450	11,140	14,850	7,910	10,540
I ½	6	1.767	1.294	12,940	16,170	22,640	13,250	17,670	9,700	12,940
I 8	$5^{\frac{1}{2}}$	2.074	1.515	15,150	18,940	26,510	15,550	20,740	11,360	15,150
1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	5	2.405	1.745	17,450	21,800	30,520	18,040	24,050	13,080	17,440
	5_	2.761	2.049	20,490	25,610	35,860	20,710	27,610	15,370	20,490
2	42	.142	2.300	23,000	28,750	40,250	23,560	31,420	17,250	23,000
$2\frac{1}{4}$ $2\frac{1}{2}$	$4\frac{1}{2}$	3.976	3.021	30,210	37,770	52,870	29,820	39,760	22,660	30,210
$2\frac{1}{2}$	4	4.909	3.716	37,160	46,450	65,040	36,820	10,000	27,870	37,160
24	4	5.940	4.620	46,200	57,750	80,840	44,580	50,100	31,650	46,200
3	$3\frac{1}{2}$	7.069	5.428	54,280	67,850	94,990	53,020	70,690	40,710	54,280

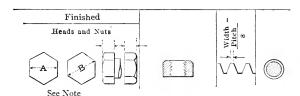
NOTE.— The distance between parallel sides of the bolt head and nut for a rough bolt is one and one-half diameters of the bolt plus one-eighth of an inch. The thickness of the head in this system for a rough bolt is equal to one-half the distance between its parallel sides. The thickness of the nut is equal to the diameter of the bolt. It was originally recommended in this system that the thickness of the bolt. If the parallel sides of the content of the parallel sides of a bolt head and nut and the thickness of the nut be one-sixteenth inch less for finished parts than for rough. However, it is the practice of bolt and nut manufacturers to make finished U. S. nuts to the same dimensions as established for rough ones, and where finished heads are required to the U. S. Standard they are customarily made to the same dimensions as rough heads unless otherwise specified.



U. S. STANDARD BOLTS AND NUTS
ROUGH

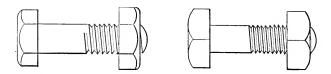
Dia. of	Threads		Across	Corners	Тпіс	KNESS	Depth of
Bolt	per Inch	Flats	В	С	Head	Nut	Thread
1/4	20	$\frac{1}{2}$	$\frac{3}{6}\frac{7}{4}$	$\frac{2}{3}\frac{3}{2}$	1/4	1	.0325
$\frac{5}{16}$	18	$\frac{1}{3}\frac{9}{2}$	$\frac{1}{16}$	$\frac{2}{3}\frac{7}{2}$	$\frac{1}{6}\frac{9}{4}$	1 5 1 6	.0361
38	16	$\frac{1}{1}\frac{1}{6}$	$\frac{51}{64}$	$\frac{3}{3}\frac{1}{2}$	$\frac{1}{3}\frac{1}{2}$	38	.0406
$\frac{7}{16}$	14	$\frac{25}{32}$	29 32	1 5 4	$\frac{25}{64}$	7 16	.0464
$\frac{1}{2}$	13	$\frac{7}{8}$	1 1/6 4	11/4	7 16	$\frac{1}{2}$.0500
9 18	12	$\frac{31}{32}$	$1\frac{1}{8}$	1 3	3 1 6 ±	9 16	.0542
$\frac{5}{8}$	11	1 1 6	$1\frac{1}{6}\frac{5}{4}$	$I\frac{1}{2}$	$\frac{1}{3}\frac{7}{2}$	<u>5</u> 8	.0590
$\frac{3}{4}$ $\frac{7}{8}$	10	$1\frac{1}{4}$	$1\frac{2}{6}\frac{9}{4}$	$1\frac{2}{3}\frac{5}{2}$	<u>5</u>	34	.0650
$\frac{7}{8}$	9	1 7 5	$1\frac{4}{6}\frac{3}{4}$	$2\frac{1}{32}$	$\frac{2}{3}\frac{3}{2}$	78	.0722
I	8	I 5/8	1 7 8	$2\frac{1}{6}\frac{9}{4}$	$\frac{1}{1}\frac{3}{6}$	1	.0812
1 1/8	7	$1\frac{1}{1}\frac{3}{6}$	$2\frac{3}{32}$	$2\frac{9}{16}$	$\frac{29}{32}$	I 1/8	.0928
$I\frac{1}{4}$	7	2	$2\frac{5}{16}$	$2\frac{5}{6}\frac{3}{4}$	I	I 1/4	.0928
1 3	6	$2\frac{3}{16}$	$2\frac{1}{3}\frac{7}{2}$	$3\frac{3}{3}\frac{3}{2}$	$1\frac{3}{32}$	I 3/8	.1083
$1\frac{1}{2}$	6	2 3 8	$2\frac{3}{4}$	3 6 4	$1\frac{3}{16}$	I 1/2	.1083
$1\frac{3}{4}$	5	$2\frac{3}{4}$	3 1 6	3 5 7 4	I 3/8	1 3/4	.1300
2	$4\frac{1}{2}$	31/8	$3\frac{3}{6}\frac{9}{4}$	$4\frac{27}{64}$	I -96	2	.1444
$2\frac{1}{4}$	$4\frac{1}{2}$	$3\frac{1}{2}$	4 3 4	$4\frac{61}{64}$	I 3/4	$2\frac{1}{4}$.1444
$2\frac{1}{2}$	4	$3\frac{7}{8}$	$4\frac{3}{6}\frac{1}{4}$	$5\frac{3}{6}\frac{1}{4}$	$I_{1}^{1}\frac{5}{6}$	$2\frac{1}{2}$.1625
$2\frac{3}{4}$	4	$4\frac{1}{4}$	$4\frac{2}{3}\frac{9}{2}$	6-1	$2\frac{1}{8}$	$2\frac{3}{4}$.1625
3	$3\frac{1}{2}$	$4\frac{5}{8}$	$5\frac{1}{3}\frac{1}{2}$	$6\frac{1}{3}\frac{7}{2}$	$2\frac{5}{16}$	3	.1857

Note. — U. S. Government Standard Bolts and Nuts are made to above U. S. or Sellers' Standard Rough Dimensions. The sizes of finished bolt heads and nuts are the same as the sizes of the rough ones, that is for finished work the forgings must be larger than for rough, thus the same wrench may be used on both black and finished heads and nuts.



U. S. STANDARD BOLTS AND NUTS. — FINISHED HEADS AND NUTS FINISHED HEADS AND NUTS

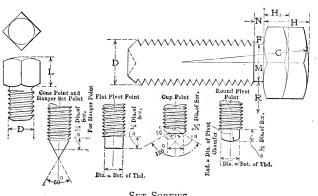
	SEE NO	TE	1	٠,	7		44	· · · >
Dia. of Bolt	Across Flats	Across Corners	Thickness	Exact Size of Hole	Tap Drill Used	Width of Flat	Area at Root of Thread	Safe Strain in lbs. Iron at 50,000 lbs. per Sq. In. Factor of Safety
14	7 16	$\frac{1}{2}$	$\frac{3}{16}$.185	.191	.0063	.0260	260
<u>5</u>	$\frac{1}{3}\frac{7}{2}$	8 9 6 4	$\frac{1}{4}$.2408	.246	.0069	.0452	452
38	<u>5</u>	$\frac{2}{3}\frac{3}{2}$	$7\frac{5}{6}$.2938	$\frac{1}{6}\frac{9}{4}$.0078	.0677	677
7 16	$\frac{2}{3}\frac{3}{2}$	$\frac{5}{6}\frac{3}{4}$	38	-3447	$\frac{2}{6}\frac{3}{4}$.0089	.0932	932
$\frac{1}{2}$	$\frac{13}{16}$	$\frac{15}{16}$	7 16	.4001	$\frac{1}{3}\frac{3}{2}$.0096	.1257	1257
T 6	$\frac{29}{32}$	1 6 1	$\frac{1}{2}$.4542	$\frac{1}{3}\frac{5}{2}$.0104	.1620	1620
· 58 34 78	1	$1\frac{5}{32}$	$\frac{9}{16}$.5069	$\frac{3}{6}\frac{3}{4}$.0114	.2018	2018
$\frac{3}{4}$	$1\frac{3}{16}$	$1\frac{2}{6}\frac{1}{4}$	$\frac{1}{1}\frac{1}{6}$.6201	<u>5</u> .	.0124	.3020	3020
78	I 3/8	$1\frac{1}{3}\frac{9}{2}$	$\frac{1}{1}\frac{3}{6}$.7307	$\frac{47}{64}$.0139	.4194	4194
1	I 7 6	$1\frac{1}{1}\frac{3}{6}$	$\tfrac{15}{16}$.8376	$\frac{2}{3}\frac{7}{2}$.0156	.5509	5509
I 1/8	1 3	. 2 1/64	$1\frac{1}{16}$.9394	$\frac{61}{64}$.0179	.6930	6930
$1\frac{1}{4}$	$1\frac{1}{1}\frac{5}{6}$	$2\frac{1}{6}\frac{5}{4}$	$1\frac{3}{16}$	1.0644	$1\frac{5}{64}$.0179	.8890	8890
1 3	2 1/8	$2\frac{2}{6}\frac{9}{4}$	$1\frac{5}{16}$	1.1585	$1\frac{1}{6}\frac{1}{4}$.0208	1.054	10540
$1\frac{1}{2}$	$2\frac{5}{16}$	2 \frac{4}{6} \frac{3}{4}	$1\frac{7}{16}$	1.2835	$1\frac{1}{6}\frac{9}{4}$.0208	1.293	12930
$1\frac{3}{4}$	$2\frac{1}{1}\frac{1}{6}$	$3\frac{7}{64}$	$1\frac{1}{1}\frac{1}{6}$	1.4902	$1\frac{3}{6}\frac{3}{4}$.0250	1.744	17440
2	$3\frac{1}{16}$	$3\frac{1}{3}\frac{7}{2}$	$1\frac{1}{1}\frac{5}{6}$	1.7113	$1\frac{2}{3}\frac{3}{2}$.0278	2.3	23000
$2\frac{1}{4}$	$3\frac{7}{16}$	$3\frac{3}{3}\frac{1}{2}$	$2\frac{3}{16}$	1.9613	$1\frac{3}{3}\frac{1}{2}$.0278	3.021	30210
$2\frac{1}{2}$	$3\frac{1}{1}\frac{3}{6}$	$4\frac{1}{3}\frac{3}{2}$	$2\frac{7}{16}$	2.1752	$2\tfrac{3}{16}$.0313	3.714	37140
$2\frac{3}{4}$	4 1 6	$4\frac{2}{3}\frac{7}{2}$	$2\frac{1}{1}\frac{1}{6}$	2.4252	$2\frac{7}{16}$.0313	4.618	46180
3 ·	4 1 6	$5\frac{3}{32}$	$2\frac{15}{16}$	2.6288	$2\frac{4}{6}\frac{1}{4}$.0357	5.427	54270



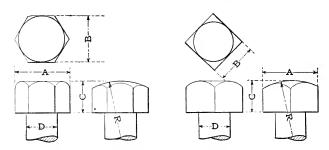
MACHINE BOLTS WITH MANUFACTURERS STD. HEADS

	Inch	Hex. AN (National	D SQUARE al Machine	Heads ry Co.)	НЕ	x. and S	QUARE N	Nuts
Dia. of Bolt	No. of Threads per Inch	Across Flats Square Heads	Across Flats Hex. Heads	Thickness Sq. and Hex. Heads	Across Flats Hex. and Sq.	Across Comers Hex. Nut	Across Corners Square Nut	Thickness Hex. and Sq.
1/4	20	3 8	7 16	3 16	- '7 16	$\frac{1}{2}$	<u>5</u> 8	3 16
$\frac{5}{16}$	18	$\frac{1}{3}\frac{5}{2}$	$\frac{1}{3}\frac{7}{2}$	$\frac{1}{4}$	$\frac{1}{3}\frac{7}{2}$	$\frac{39}{64}$	$\frac{3}{4}$	$\frac{1}{4}$
$\frac{3}{8}$	16	$\frac{9}{16}$	<u>5</u>	$\frac{9}{32}$	<u>5</u> 8	$\frac{2}{3}\frac{3}{2}$	$\frac{5}{6}\frac{7}{4}$	5 16
7 16	14	$\frac{2}{3}\frac{1}{2}$	$\frac{2}{3}\frac{3}{2}$	$\frac{2}{6}\frac{1}{4}$	$\frac{2}{3}\frac{3}{2}$	$\frac{53}{64}$	I	3 8
$\frac{1}{2}$	13	34	18 16	<u>3</u> 8	$\frac{13}{16}$	$\frac{15}{16}$	$1\frac{5}{32}$	$\frac{7}{16}$
$\frac{9}{1}\overline{6}$	12	$\frac{27}{32}$	$\frac{2}{3}\frac{9}{2}$	$\frac{2}{6}\frac{7}{4}$	$\frac{29}{32}$	1 1/6 4	$1\frac{9}{32}$	$\frac{1}{2}$
<u>5</u>	11	15 16	I	$\frac{1}{3}\frac{7}{2}$	I	$1\frac{5}{32}$	$1\frac{27}{64}$	9 16
5 8 3 4 7 8	10	I 1/8	$1\frac{3}{16}$	9 16	I 3 6	$1\frac{2}{6}\frac{1}{4}$	$1\frac{2}{3}\frac{1}{2}$	$\frac{1}{1}\frac{1}{6}$
$\frac{7}{8}$	9	$1\frac{5}{16}$	1 3	$\frac{2}{3}\frac{1}{2}$	I 3/8	$1\frac{1}{3}\frac{9}{2}$	$1\frac{1}{1}\frac{5}{6}$	$\frac{1}{1}\frac{3}{6}$
I	8	$I\frac{1}{2}$	$1\frac{9}{16}$	34	$1\frac{9}{16}$	$1\frac{1}{1}\frac{3}{6}$	$2\frac{1}{1}\frac{3}{6}$	$\frac{15}{16}$
$I\frac{1}{8}$	7	$1\frac{1}{1}\frac{1}{6}$	$1\frac{3}{4}$	$\frac{27}{32}$	$1\frac{1}{1}\frac{3}{6}$	$2\frac{3}{32}$	$2\frac{9}{16}$	$1\frac{1}{8}$
$1\frac{1}{4}$	7	1 7/8	$1\frac{1}{1}\frac{5}{6}$	$\frac{15}{16}$	2	$2\frac{5}{16}$	$2\frac{5}{6}\frac{3}{4}$	$I_{\frac{1}{4}}$
$1\frac{3}{8}$	6	$2\frac{1}{16}$	$2\frac{1}{8}$	$I_{\frac{1}{16}}$	$2\frac{3}{16}$	$2\frac{1}{3}\frac{7}{2}$	$3\frac{3}{32}$	$1\frac{3}{8}$
$I\frac{1}{2}$	6	21/4	$2\frac{5}{16}$	I 1/8	2 3 /8	$2\frac{3}{4}$	$3\frac{2}{6}\frac{3}{4}$	$I\frac{1}{2}$
$1\frac{5}{8}$	$5^{\frac{1}{2}}$	$2\frac{7}{16}$	$2\frac{1}{2}$	$1\frac{7}{32}$	$2\frac{9}{16}$	$2\frac{3}{3}\frac{1}{2}$	$3\frac{5}{8}$	I 5/8
$I\frac{3}{4}$	5	2 5	$2\frac{1}{1}\frac{1}{6}$	$1\frac{5}{16}$	$2\frac{3}{4}$	$3\frac{3}{16}$	$3\frac{5}{6}\frac{7}{4}$	$1\frac{3}{4}$
$I\frac{7}{8}$. 5	$2\frac{13}{16}$	2 7/8	$1\frac{1}{3}\frac{3}{2}$	$2\frac{1}{1}\frac{5}{6}$	$3\frac{1}{3}\frac{3}{2}$	$4\tfrac{5}{3}\tfrac{5}{2}$	$1\frac{7}{8}$
2	$4\frac{1}{2}$	3	316	$1\frac{1}{2}$	31/8	3 8 9	464	2

 $\rm Note.-Nuts$ supplied by different makers for manufacturers standard bolts vary somewhat as regards thickness. The above nut sizes are Hoopes and Townsend Standard.

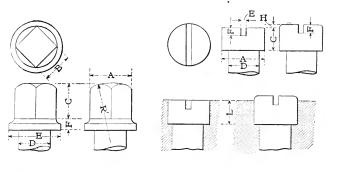


				Set S	Screws				
				Hartfori	о Масні	ne Scre	w Co.	Standar	D
Dia. of Screw	No. of Threads . per Inch	Length of Head	Length of Head	Length of Low Head	Radius of Crown	Dia. of Neck	Radius of Neck	Fillet — Neck	Length of Neck
D		L	Н	H ₁	С	М	R	F	N
14	20	14	7 3 2	1/8	<u>5</u>	7/3/2	1/4	.019	.075
$\frac{5}{16}$	18	7 <u>5</u>	$\frac{35}{128}$	$\frac{5}{32}$	$\frac{25}{32}$	$\frac{35}{128}$	1 ⁵ 6	.021	.083
38	16	3 8	$\frac{21}{64}$	3 1 6	$\frac{1}{1}\frac{5}{6}$	21 64	38	.023	.094
$\frac{7}{16}$	14	7 16	$\frac{49}{128}$	3 ⁷ / ₂	1 3 2	$\begin{array}{c} 4.9 \\ \overline{128} \end{array}$	7 16	.027	.107
$\frac{1}{2}$	13	$\frac{1}{2}$	7 16	1	I 1/4	7 16	$\frac{1}{2}$.031	.125
$\frac{9}{16}$	12	1 g	$\frac{63}{128}$	$\frac{9}{32}$	$1\frac{1}{3}\frac{3}{2}$	$\frac{63}{128}$	7 6 T 6	.032	.125
<u>5</u>	11	<u>5</u>	$\frac{3}{6}\frac{5}{4}$	5 16	$1\frac{9}{16}$	$\frac{3}{6}\frac{5}{4}$	5 8	.03.4	.130
3 4	10	$\frac{3}{4}$	$\frac{21}{32}$	3 8	I 7 8	$\frac{21}{32}$	$\frac{3}{4}$.037	.150
78	9	78	$\frac{49}{64}$	7 16	$2\frac{3}{16}$	$\tfrac{49}{64}$	78	.041	.166
1	•8	I	7/8	$\frac{1}{2}$	$2\frac{1}{2}$	78	I	.047	.187

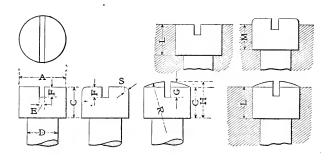


HARTFORD MACHINE SCREW CO., STANDARD

	HEXA	GON HE	EAD CAE	Screws	3	Squa	RE HEAL	CAP Sc	REWS
Dia. of Screw	No. of Threads per In.	Distance Across Corners	Distance Across Flats	Thickness of Head	Radius of Head	Distance Across Corners	Distance Across Flats	Thickness of Head	Radius of Head
D		A	В	С	R	A	В	С	R
1/4	20	$\frac{1}{2}$	7 1 6	1/4	34	$\frac{1}{3}\frac{7}{2}$	38	1 4	$\frac{25}{32}$
16	18	3 7 6 4	$\frac{1}{2}$	$\frac{5}{16}$	$\frac{5}{6}\frac{5}{4}$	<u>5</u> 8	7 16	5 16	$\frac{15}{16}$
38	16	$\frac{2}{3}\frac{1}{2}$	$\frac{9}{16}$	38	31	$\frac{45}{64}$	$\frac{1}{2}$	3 8	1 1 6
$\frac{7}{16}$	14	$\frac{23}{32}$	<u>5</u>	$\frac{7}{16}$	$1\frac{5}{64}$	$\frac{5}{6}\frac{1}{4}$	9 16	$\frac{7}{16}$	1 3/6
$\frac{1}{2}$	13	7 8	34	$\frac{1}{2}$	$1\frac{9}{32}$	5 7 6 4	<u>5</u>	$\frac{1}{2}$	$1\frac{5}{16}$
$\frac{9}{16}$	12	$\frac{15}{16}$	$\frac{1}{1}\frac{3}{6}$	9 16	$1\frac{1}{3}\frac{3}{2}$	$\frac{3}{3}\frac{1}{2}$	$\frac{1}{1}\frac{1}{6}$	<u>9</u> 16	$1\frac{1}{3}\frac{5}{2}$
<u>5</u>	11	1 1 4	78	<u>5</u>	$1\frac{1}{2}$	$1\frac{1}{16}$	34	<u>5</u>	$1\frac{19}{32}$
34	10	$1\frac{5}{32}$	Ι.	34	$1\frac{2}{3}\frac{3}{2}$	$1\frac{15}{64}$	78	34	$1\frac{27}{32}$
78	9	I 1 9 4	I 1/8	7 8	$1\frac{15}{16}$	$1\frac{1}{3}\frac{9}{2}$	I 1/8	78	2 3 8
I	8	1 7/6	I 1/4	1	$2\frac{5}{32}$	$1\tfrac{49}{64}$	11/4	ı	$2\frac{2}{3}\frac{1}{2}$
$I\frac{1}{8}$	7	$1\frac{1}{3}\frac{9}{2}$	1 3/8	I 1/8	2 3 8	$1\frac{15}{16}$	13/8	I 1/8	3
I 1/4	7	$1\frac{47}{64}$	1 1/2	11/4	$2\frac{19}{32}$	2 1 /8	$I^{\frac{1}{2}}$	11/4	3 1 6

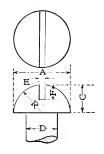


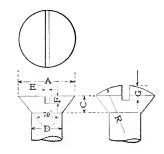
	Col	LAR I	HEAL	Sci	REWS				Filli	STER (P	Не.	AD CAP So W. St'd)	CREW	s
Dia. of Screw	No. of Threads per Inch	Distance Across Corners	Distance Across Flats	Length of Head	Dia. of Collar	Thickness of Collar	Radius of Head	Dia. of Head	Length of Flat Head	Hight of Round Corner	Length of Round Head	Width of Slot	Depth of Slot	Depth of Coun- terbore
D		A	В	С	Е	F	R	A	С	Н	C+ H	Е	F	L
1/8	40	$\frac{1}{6}\frac{1}{4}$	1/8	9 64	1/4	1 16	$\frac{1}{4}$	11 64	$\frac{3}{32}$	1 3 2	18	.025	$\frac{1}{32}$	3 3 2
$\frac{3}{16}$	32	$\frac{17}{64}$	3 16	$\frac{7}{32}$	$\frac{1}{3}\frac{1}{2}$	5 64	38	1/4	1/8	$\frac{1}{32}$	$\frac{5}{32}$.039	6 ³ 4	1/8
$\frac{1}{4}$	20	$\frac{1}{3}\frac{1}{2}$	14	$\frac{9}{32}$	176	$\frac{3}{32}$	$\frac{1}{2}$	$\frac{1}{3}\frac{1}{2}$	$\frac{5}{32}$	$\frac{1}{32}$	3 16	.058	5 6 1	$\frac{5}{32}$
$\frac{5}{16}$	18	T 6	$\frac{5}{16}$	$\frac{1}{3}\frac{1}{2}$	$\frac{1}{2}$	1/8	$\frac{1}{1}\frac{1}{6}$	7 6	3 16	$\frac{3}{64}$	$\frac{1}{6}\frac{5}{4}$.071	$\frac{5}{64}$	$\frac{3}{16}$
38	16	$\frac{1}{3}\frac{7}{2}$	38	$\frac{1}{3}\frac{3}{2}$	<u>5</u>	$\frac{5}{32}$	$\frac{13}{16}$	$\frac{1}{2}$	14	$\frac{3}{64}$	$\frac{19}{64}$.086	$\frac{7}{64}$	· 1/4
$\frac{7}{16}$	14	<u>5</u>	$\frac{7}{16}$	$\frac{1}{3}\frac{5}{2}$	$\frac{1}{1}\frac{1}{6}$	$\frac{3}{16}$	$\frac{15}{16}$	9 16	1 4	$\frac{3}{64}$	19 64	.0 99	6 4	$\frac{1}{4}$
$\frac{1}{2}$	13	$\frac{45}{64}$	$\frac{1}{2}$	1 6	13 16	$\frac{1}{6}\frac{3}{4}$	$1\frac{1}{16}$	$\frac{11}{16}$	$\frac{5}{16}$	$\frac{3}{64}$	$\frac{2}{6}\frac{3}{4}$.I I 2	1/8	$\frac{5}{16}$
$\frac{9}{16}$	12	$\frac{5}{6}\frac{1}{4}$	$\frac{9}{16}$	<u>5</u>	$\frac{15}{16}$	$\frac{7}{32}$	$1\frac{3}{16}$	34	<u>3</u>	$\frac{3}{64}$	$\frac{2}{6}\frac{3}{4}$.133	9 64	38
<u>5</u>	11	$\frac{5}{6}\frac{7}{4}$	<u>5</u>	$\frac{1}{1}\frac{1}{6}$	ı	$\frac{1}{4}$	$1\frac{5}{16}$	$\frac{7}{8}$	$\frac{7}{16}$	$\frac{1}{16}$	1/2	.133	5 32	7 1 6
3 4	10	1 1 6	3	$\tfrac{13}{16}$	14	5 16	1 9 6	I 1 6	$\frac{1}{2}$	1 6	9	.133	$\frac{3}{16}$	$\frac{1}{2}$
78	9							$1\frac{3}{16}$	9 16	1 6	<u>5</u>	.133	$\frac{7}{32}$	$\frac{9}{16}$
I	8							I 3/8	58	16	$\frac{1}{1}\frac{1}{6}$.165	1/4	<u>5</u> 8



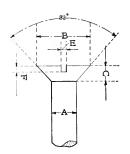
FLAT, ROUND AND OVAL FILLISTER HEAD CAP SCREWS

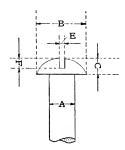
Dia. of Screw	No. of Threads per Inch	Dia. of Head	Length of Flat and Round Head	Radius of Corner	Length of Oval Head	Radius of Crown	Width of Slot	Depth of Slot	Depth of Slot Oval Head	Depth of Counter Bore, Flat and Oval Head	Depth of Coun- terbore Round Head
D		A	С	s	н	R	Е	F	G	L	M
1/8	40	3 1 6	1 8	1 3 2	9 6 4	14	.032	1 16	54	1/8	$\frac{3}{32}$
$\frac{3}{16}$	32	14	3 16	$\frac{1}{32}$	$\frac{7}{32}$	5 16	.040	$\frac{1}{16}$	3 2,	$\frac{3}{16}$	$\frac{5}{32}$
$\frac{1}{4}$	20	38	1/4	$\frac{1}{32}$	$\frac{9}{32}$	$\frac{1}{2}$.064	1 16	$\frac{3}{32}$	1/4	$\frac{7}{3}$ 2
$\frac{5}{16}$	18	<u>1</u> 76	15 16	$\frac{1}{16}$	$\frac{2}{6}\frac{3}{4}$	<u>5</u>	.072	5 6 4	18	15 16	$\frac{1}{4}$
38	16	$\frac{9}{16}$	<u>3</u>	$\frac{1}{16}$	$\frac{1}{3}\frac{5}{2}$	34	.091	$\frac{3}{32}$	$\frac{9}{64}$	38	$\frac{5}{1}$ $\overline{6}$
$\Gamma^{7}_{\overline{6}}$	14	<u>5</u>	7 16	$\frac{1}{16}$	$\frac{1}{2}$	- 7 8	.102	<u>-</u> 7 _∓	$\frac{1}{6}\frac{1}{4}$	$\frac{7}{16}$	<u>3</u>
$\frac{1}{2}$	13	$\frac{3}{4}$	$\frac{1}{2}$	$\frac{1}{16}$	$\frac{9}{16}$	$1\frac{1}{16}$.114	1/8	$\frac{3}{16}$	$\frac{1}{2}$	76
$\frac{9}{16}$	I 2	$\frac{1}{1}\frac{3}{6}$	9 16	$\frac{1}{16}$	$\frac{4}{6}\frac{1}{4}$	1 ½	.114	9 64	$\frac{7}{32}$	$\frac{9}{16}$	$\frac{1}{2}$
<u>5</u>	11	7/8	<u>5</u>	$\frac{1}{16}$	$\tfrac{4}{6} \tfrac{5}{4}$	I 1/4	.128	$\frac{5}{32}$	$\frac{1}{6}\frac{5}{4}$	<u>5</u>	$\frac{9}{16}$
$\frac{3}{4}$	10	1	34	$\frac{1}{16}$	$\frac{2}{3}\frac{7}{2}$	$1\frac{1}{2}$.133	$\frac{3}{16}$	$\frac{9}{32}$	34	$\frac{1}{1}\frac{1}{6}$
7 8	9	1 1/8	7 8	$\frac{1}{16}$	$\frac{6}{6}\frac{3}{4}$	I 5/8	.133	$\frac{7}{32}$	$\frac{2}{6}\frac{1}{4}$	78	$\frac{1}{1}\frac{3}{6}$
1	8	11	I	$\frac{3}{3}$	I 1/8	13	.165	14	38	I	$\frac{29}{32}$





	Bur	ron H	EAD (CAP So	CREWS		FLAT	AND C	OVAL C CAP S	OUNTERS CREWS	UNK I	TEAD
Dia. of Screw	No. of Threads per Inch	Dia. of Head	Length of Head	Radius of Head	Width of Slot	Depth of Slot	Dia. of Head	Length of Flat Head	Radius of Crown	Width of Slot	Depth of Slot Flat Head	Depth of Slot Oval Head
D		A	С	R	Е	F	A	С	R	E	F	G
18	40	$\frac{7}{32}$	₹ 64	7 6 4	.035	3 64	14	$\frac{3}{32}$	<u>3</u> 8	.040	: <u>3</u>	3 64
$\frac{3}{16}$	32	$\frac{5}{16}$	$\frac{5}{32}$	$\frac{5}{32}$.051	.16	38	9 64	9 1 6	.064	1 16	1 6
$\frac{1}{4}$	20	$\frac{7}{16}$	$\frac{7}{32}$	$\frac{7}{32}$.072	5 64	$\frac{15}{32}$	$\frac{5}{32}$	$\frac{29}{32}$.072	5 6 4	5 64
$\frac{5}{16}$	18	$\frac{9}{16}$	$\frac{9}{32}$	$\frac{9}{32}$.091	$\frac{3}{32}$	<u>5</u>	7/3/2	$\frac{1}{1}\frac{5}{6}$.102	3 2	3 2
<u>3</u>	16	<u>5</u>	5 16	5 16	.102	₹ 64	34	17 64	I 1/8	.114	5 64	6 4
$\frac{7}{16}$	1.4	$\frac{3}{4}$	38	<u>3</u>	.114	8	$\frac{1}{1}\frac{3}{6}$	$\frac{17}{64}$	$1\frac{7}{32}$.114	18	1/8
$\frac{1}{2}$	13	$\begin{array}{c} \frac{1}{1}\frac{3}{6} \end{array}$	$\frac{1}{3}\frac{3}{2}$	$\frac{1}{3}\frac{3}{2}$.114	$\frac{5}{32}$	7 8	$\frac{1}{6}\frac{7}{4}$	$1\frac{5}{16}$.128	<u>5</u> 3 2	$\frac{5}{32}$
$\frac{9}{16}$	12	$\tfrac{15}{16}$	$\frac{1}{3}\frac{5}{2}$	$\frac{15}{32}$.114	$\frac{1}{6}\frac{1}{4}$	1	5 1 6	$1\frac{1}{2}$.133	$\frac{1}{6}\frac{1}{4}$	$\frac{1}{6}\frac{1}{4}$
$\frac{5}{8}$	11	I	$\frac{1}{2}$	$\frac{1}{2}$.133	3 1 6	1 1/8	$\frac{23}{64}$	$I\frac{1}{1}\frac{1}{6}$.133	1 8 €	$\frac{3}{16}$
3 4	10	I 1/4	<u>5</u> 8	<u>5</u> 8	.133	3 2	I 3/8	7 16	$2\frac{1}{16}$.133	32	372

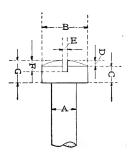




MACHINE SCREWS. AMERICAN SCREW COMPANY

NT.	A		FLAT I	TEAD			ROUND	HEAD	
No.	A	В	С	Е	F	В	С	E	F
2	.0842	.1631	.0454	.030	.0151	.1544	.0672	.030	.0403
3	.0973	.1894	.0530	.032	.0177	.1786	.0746	.032	.0448
4	.1105	.2158	.0605	.034	.0202	.2028	.0820	.034	.0492
5	.1236	.2421	.0681	.036	.0227	.2270	.0894	.036	.0536
6	.1368	.2684	.0757	.039	.0252	.2512	.0968	.039	.0580
7	.1500	.2947	.0832	.041	.0277	.2754	.1042	.041	.0625
8	.1631	.3210	.0908	.043	.0303	.2996	.1116	.043	.0670
9	.1763	.3474	.0984	.045	.0328	.3238	.1190	.045	.0714
10	.1894	-3737	.1059	.048	.0353	.3480	.1264	.048	.0758
12	.2158	.4263	.1210	.052	.0403	.3922	.1412	.052	.0847
14	.2421	.4790	.1362	.057	.0454	.4364	.1560	.057	.0936
16	.2684	.5316	.1513	.061	.0504	.4806	.1708	.061	.1024
18	.2947	.5842	.1665	.066	.0555	.5248	.1856	.066	.1114
20	.3210	.6368	.1816	.070	.0605	.5690	.2004	.070	.1202
22	•3474	.6895	.1967	.075	.0656	.6106	.2152	.075	.1291
24	-3737	.7421	.2118	.079	.0706	.6522	.2300	.079	.1380
26	.1000	.7421	.1967	.084	.0656	.6938	2448	.084	.1469
28	.4263	.7948	.2118	.088	.0706	.7354	.2596	.088	.1558
30	.4526	.8474	.2270	.093	.0757	.7770	.2744	.093	.1646

Dimensions given are maximum, the necessary working variations being below them.



MACHINE SCREWS. AMERICAN SCREW COMPANY

No.	A		Fili	LISTER H	EAD		G				
No.	A	В	С	D	Е	F	G				
2	.0842	.1350	.0549	.0126	.030	.0338	.0675				
3	.0973	.1561	.0634	.0146	.032	.0390	.0780				
4	.1105	.1772	.0720	.0166	.034	.0443	.0886				
5	.1236	.1984	.0806	.0186	.036	.0496	.0992				
6	.1368	.2195	.0892	.0205	.039	.0549	.1097				
7	.1500	.2406	.0978	.0225	.041	.0602	.1203				
8	.1631	.2617	.1063	.0245	.043	.0654	.1308				
9	.1763	.2828	.1149	.0265	.045	.0707	.1414				
10	.1894	.3040	.1235	.0285	.048	.0760	.1520				
I 2	.2158	.3462	.1407	.0324	.052	.0866	.1731				
14	.2421	.3884	.1578	.0364	.057	.0971	.1942				
16	.2684	.4307	.1750	.0403	.061	.1077	.2153				
18	•2947	.4729	.1921	.0443	.066	.1182	.2364				
20	.3210	.5152	.2093	.0483	.070	.1288	.2576				
22	•3474	-5574	.2267	.0520	.075	.1384	.2787				
24	·3737	.5996	.2436	.0562	.079	.1499	.2998				
26	.4000 ·	.6419	.2608	.0601	.084	.1605	.3209				
28	.4263	.6841	.2779	.0641	.088	.1710	.3420				
30	.4526	.7264	.2951	.0681	.093	.1816	.3632				

AMERICAN	SCRE	W C	OMPA1	NY. 2	STAND	ARD	1 HRE	ADS	PER	INCH			
No.	2	3	4	5	6	7	8	9	IO	12			
Threads per Inch	48 56 64	48 56	3 ² ,	36 .0	30 32 36	30 32	30 32 36		30, 32	20 24			
No.		14	16	18	20	22	24	26	28	30			
Threads pe	r	18 20. 24		18,	16,	18	14 16 18		14, 16				

A.S.M.E. STANDARD PROPORTIONS OF MACHINE SCREWS

THE diagram and tables herewith show the proportions of machine screws as recommended by the committee of the American Society of Mechanical Engineers on Standard Proportions for Machine Screws, the report of this committee being adopted by the Society

at its spring meeting, 1907.

The included angle is 60 degrees, and the flat at top and bottom of thread is one eighth of the pitch for the basic or standard diameter. There is a uniform increment of 0.013 inch, between all sizes from 0.06 to 0.19 (numbers 0 to 10 in the tables which follow) and of 0.026 inch in the remaining sizes. This change has been made in the interest of simplicity and because the resulting pitch diameters are more nearly in accord with the pitch diameters of screws in present use.

The pitches are a function of the diameter as expressed by the

formula

Threads per inch =
$$\frac{6.5}{D + 0.02}$$

with the results given approximately, so as to avoid the use of fractional threads.

The diagram shows the various sizes for both 16 and 72 threads per inch, and shows, among other things, the allowable difference in the flat surface, between the maximum tap and the minimum screw, this variation being from one-eighth to one sixteenth.

The minimum tap conforms to the basic standard in all respects, except diameter. The difference between the minimum tap and the maximum screw provides an allowance for error in pitch and

for wear of tap in service.

The form of tap thread shown is recommended as being stronger and more serviceable than the so-called V-thread, but as some believe a strict adherence to the form shown might add to the cost of small taps, they have decided that taps having the correct angle and pitch diameter are permissible even with the V-thread. This will allow a large proportion of the taps now in stock to be utilized.

The tables given by the committee were combined into the present

compact form by the Corbin Screw Corporation.

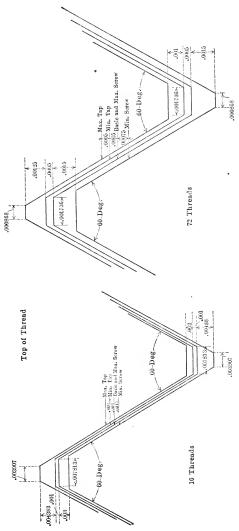


Diagram Showing Form of Basic Maximum and Minimum Screw and Tap Threads A. S. M. E. Machine Screw Standard

A. S. M. E. STANDARD MACHINE SCREWS

	Size	OUT	Outside Diameters	ERS	Pn	PITCH DIAMETERS	RS	Rc	Root Diameters	RS
No.	Out. Dia. and Thds. P. I.	Minimum	Maximum	Difference	Minimum	Maximum	Difference	Minimum	Maximum	Difference
c	0000	0 5 7 3 0	090	8600	1 (1			0.1.	0
) -	0 0 0	2/60.	0 0	02000	5000	6160.	+100.	01400	.0430	.0020
-	2/2/2	0/0.	5/5	.003	.0025	t00.	.0015	.052	.055	.0030
61	to-080·	.0828	.080	.0032	.0743	.0759	0100.	.0624	.0657	.0033
3	05-660.	.0955	660.	.0035	.0857	.0874	7100.	.0721	.0758	.0037
4	.112-48	.1082	.112	.0038	9960.	.0985	0100.	.0807	.0849	.0042
rv.	.125-44	.1210	.125	0000	.1082	.1102	.0020	01()0,	.0955	.0045
9	.138-40	.1338	.138	.0042	7611.	.1218	.002 I	1001.	.1055	.0048
-	.151-36	.1466	.151	++00.	.1308	.1330	.0022	2601.	6411.	.0052
×	.164-36	.1596	.164	t+00·	.1438	.146	.0022	.1227	.1279	.0052
6	.177-32	.1723	.177	2100.	.1544	.1567	.0023	.1307	.1364	.0057
IO	.190-30	.1852	061.	stoo.	991.	.1684	.0024	.1407	.1467	0900*
12	.210-28	.2111	.216	61000	+061.	8261.	.0024	.1633	. 9691.	.0063
14	.242-24	.2368	.242	.0052	.2123	.2149	9200.	8081.	6281.	1700.
91	.268-22	.2626	.268	.0054	.2358	.2385	.0027	.2014	.209	9200.
18	.294-20	.2884	.294	9500.	.2587	.2615	.0028	.2208	.229	.0082
20	.320-20	.3144	.320	9500*	.2847	.2875	.0028	.2468	.255	.0082
22	.340-18	.3402	.346	.0058	.3070	.3099	6200.	.2649	.2738	6800.
24	.372-16	.306	.372	0900.	.3284	.3314	.0030	.281	.2908	8600.
50	.398-16	.392	.398	0900.	.3544	.3574	.0030	.307	3168	8600.
28	.424-14	.4178	.424	2900.	.3745	.3776	.0031	.3204	.3312	8010.
30	.450-14	.4438	.450	2900.	.4005	.4036	.0031	.3464	-3572	8010.

TAPS FOR A. S. M. E. STANDARD MACHINE SCREWS

ap Drill	Diameters	.0465	.0505	020	.0785	080	.0995	011.	.120	.136	1405	.152	.173	.1935	.213	.234	.261	.281	.2968	.323	.339	.368
		0100.	.0020	.0021	.0023	.0025	.0027	.0028	.0020	.0029	.0031	.0032	.0033	.0035	.0036	.0037	.0037	.0039	0100.	0000.	Itoo.	1100.
Root Diameters	Maximum Difference Minimum Maximum Difference	.0466	.058	6890.	.0793	.0887	.0995	7601.	.1193	.1323	11411	.1515	.1745	.1932	.2144	.2346	.2606	.2796	.2968	.3228	.3374	.3034
Roo	Minimum	7+40-	950	8990	.077	.0862	8960.	6901.	1911.	.1294	.138	.1483	.1712	7681.	.2108	.2309	.2569	.2757	.2928	.3188	-3333	.3593
ERS	Difference	100.	100.	1100°	1100.	.0012	.0013	+100°	+100·	.0014	.0015	9100.	.7100.	7100.	8100.	8100.	8100.	.0020	.0020	.0020	.0021	.0021
PITCH DIAMETERS	Maximum	.0538	990.	.0781	7080.	101.	.1129	.1246	.1359	.1480	8651.	.1716	1961.	.2184	1242.	.2652	2162.	.3138	-3354	.3614	.3818	3/ot-
	Minimum	.0528	590.	0770.	9880.	86600	91111	.1232	.1345	.1475	.1583	.170	.1944	.2167	.2403	.2634	+682.	.3118	-3334	.3594	.3797	.4057
TERS	Minimum Maximum Difference Minimum	.0023	.0025	.0027	.0031	.00-35	.0038	1100.	.0044	++00.	6400.	.0052	9500	.0062	2900	.0072	.0072	0800.	8800.	.0088	8600.	8600.
Outside Diameters	Maximum	,0632	.0765	8080.	.1033	.1168	.1301	.1435	.1569	6691	.1835	8961.	.2232	-250	-2765	.3031	.3291	•3559	.3828	.4088	•4359	.4619
OUTSI	Minimum	6090.	.074	.0871	.1002	.1133	.1263	1394	.1525	.1655	.1786	9161.	.2176	.2438	8692.	.2959	.3219	-3479	-374	001:	.4261	.452I
Size	Out. Dia. and Thds. P. In.	08-090.	.073-72	t9-980°	95–660°	.112-48	.125-44	01-38-10	.151-36	.164-36	.177-32	.190-30	.216-28	.242-24	.268-22	.294-20	.320-20	.346-18	.372-16	.398-16	.424-14	.450-14
	No.	0	H	8	3	4	Ŋ	9	7	∞	6	OI	12	14	91	18	50	22	24	56	28	30

SCREW
MACHINE
SPECIAL
Εij
M.
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292	ŀ					DC	<i>)</i>	. 1	ο,	Τ,	···	1,	.	Λı	Α Τ	,	50	,11	15	V V .	,							
RS	Difference	.0033	.0037	.0042	8400.	.0052	8400.	.0052	.0052	.0057	.0057	0900	2500.	0900*	0900	1,000	.0057	1200.	1200.	.0082	.0082	6800.	6800.	8600.	6800.	8010.	8600.	8600.
ROOT DIAMETERS	Maximum	7550.	.0628	6170.	.0795	.0759	.0925	6880.	6101.	+260.	toii.	.1077	.1234	.1207	.1337	.1229	.1494	•1359	6191.	.1770	.203	.2218	.2478	.2648	8662.	.3052	.3428	.3688
RC	Minimum	†6†o•	.0501	22000	.0747	7070.	.0877	.0837	2960	7160.	1001	.1017	.1177	7+11.	.1277	.1158	.1437	.1288	.1548	.1688	8+61.	.2129	.2389	.255	6062*	-2944	+333	.359
RS	Difference	9100	7100.	6100.	.0021	.0022	.002 I	.0022	.0022	.0023	.0023	.0024	.0023	.0024	.0024	.0026	.0023	.0026	.0026	.0028	.0028	6200*	.0029	.0030	.0029	.0031	.0030	.0030
ACHINE SCREW PITCH DIAMETERS	Maximum	.0629	.0744	.0855	.0958	+00.	.1088	701.	.120	.1177	.1307	.1294	.1437	.1424	.1553	.1499	7691.	.1629	6881.	.2095	.2355	.2579	.2839	.3054	.3359	.3516	.3834	.4094
Pm Pm	Minimum	,0613	.0727	.0836	.0937	8160.	7901.	8101.	.1178	1154	.1284	.1270	.1414	.1400	.1529	.1473	.1674	.1603	.1863	.2067	.2327	-255	.281	.3024	.333	.3485	.3804	.4064
A. S. IM. E. SPECIAL INACHINE SCREWS AMETERS PITCH DIAMETERS	Difference	20032	.0035	.0038	.00.42	1000	.0042	++00.	+600.	.0047	2000	5,00.	2000	8100.	.0048	.0052	2000	.0052	.0052	9500.	.0056	.1058	.0058	0900	.0058	2900.	0900*	0900
OUTSIDE DIAMETERS	Maximum	.073	980.	660*	.112	.112	.125	.125	.138	.138	.151	.151	+91.	+91.	.177	.177	061.	061.	.216	.242	.268	.294	.320	.346	.372	.398	.424	.450
Our	Minimum	8690	.0825	.0952	8/01.	9201.	.1208	.1206	.1336	.1333	.1463	.1462	.1593	.1592	.1722	.1718	.1853	.1848	.2108	-2364	.2624	.2882	.3142	-340	.3662	.3918	.418	+++
Size	Out. Dia. and Thds. P. I.	.073-64	.086-56	81:-660	.112-40	36	.125-40	36	.138-36	32	.151-32	30	.164-32	30	.177-30	24	.190-32	24	.216-24	.242-20	.268-20	.294-18	.320-18	.346-16	.372-18	.398-14	.424-16	.450-16
	No.	н	7	8	4		Ŋ	,	9		7		00		6		OI		12	14	91	81	20	22	24	56	28	30

Special Taps. A. S. M. E. Machine Screws

	Size	OUTS	OUTSIDE DIAMETERS	TERS	PITC	PITCH DIAMETERS	ERS	Roo	ROOT DIAMETERS		Tap Drill
.cN	Out. Dia. and Thds P In	Minimum	Maximum	Minimum Maximum Difference Minimum Maximum Difference Minimum Maximum Difference	Minimum	Maximum	Difference	Minimum	Maximum	Difference	Diameters
-	040-640	1770	8920	.0027	190.	1590	1100	.0538	.0559	.0021	.055
٠ ،	40° 5/5° 08/0-1-080	0872	0003	0031	9540	.0767	1100	*90	.0663	.0023	290.
,	8:-000	1001	1028	0037	8980	.880	.0012	.0732	.0757	.0025	920.
· ·	04-601	1127	1177	1500.	.0072	9860	,0014	0800	.0837	.0028	.082
4	95	1135	0/11	.0044	.0055	6960	\$100	.0774	.0803	.0029	180°
u	35.	1264	1305	.0041	.1102	91111	,0014	.0939	2960	.0028	860.
0	36	.1265	.1300	.0044	.1085	.1099	4100.	.0904	.0933	.0029	.0935
9	.128-26	.1305	.1439	.0044	.1215	.1229	. 0014	.1034	.1063	.0029	.1005
,	22	1306	.1445	.0040	.1193	.1208	.0015	660.	1001.	.003I	Sioi.
r	151-22	1526	577 I.	0000	.1323	.1338	.0015	.112	.1151	.0031	911.
`	20 20 20	1526	1578	.0052	•131	.1326	9100.	.1093	.1125	.0032	.113
~	161-22	9591	1705	.0040	.1453	.1468	5100.	.125	.1281	.003I	.1285
	200	.1656	.1708	.0052	144	.1456	9100.	.1223	.1255	.0032	.1285
<u> </u>	05-777.	1786	.1838	.0052	.1569	.1585	9100.	.1353	.1385	.0032	.1405
,	24	1788	.185	.0062	1517	.1534	7100.	.1247	.1282	.0035	.1285
9	100-22	9101	.1065	.0040	.1713	.1728	.0015	.151	.1541	.0031	•154
2	24	8101.	.108	.0062	.1647	.1664	7100.	.1377	.1412	.0035	.1405
. 12	72-012.	.2178	224	.0062	7001.	.1924	7100.	.1637	.1672	.0035	991.
1 5	242-20	.2430	.251I	.0072	2114	.2132	\$100.	6841.	.1826	.0037	.182
+ 4	06-800	2,500	177Z	.0072	.2374	.2392	8100.	5040	.2086	.0037	.209
2 %	204-18	.2050	.3030	0800.	.2508	2618	.0020	.2237	.2276	.0039	.228
2 6	320-18	.2210	.3200	0800.	.2858	.2878	.0020	.2497	.2536	.0039	.257
2 6	3.46-16	.348	.3568	.0088	.3074	*3001	.0020	.2668	.2708	.0040	.272
	373-18	.3730	.3810	0800.	.3378	.3398	.0020	.3017	.3056	•0039	-3125
56	308-14	1001	.4009	8000.	-3537	.3558	.002I	.3073	3114	.004I	.310
28	424-16	.426	-4348	.0088	.3854	.3874	.0020	.3448	.3488	•0040	.340
30	91-054.	.452	.4608	.0088	4114	.4134	0200.	.3708	.3748	.0040	-377

PROPORTIONS OF MACHINE SCREW HEADS A. S. M. E. Standard

THE four standard heads are given herewith. These proportions are based on and include the diameter of the screw, diameter of the head, thickness of head, width and depth of slot, radius for round and fillister heads, and included angle of the flat-head screw.

Oval Fillister Head Machine Screws. A. S. M. E. Standard

OVAL FILLISTER HEAD SCREWS

A = Diameter of Body.

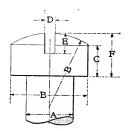
B = 1.64A - .009 = Diam. of Head and Rad. for Oval

C = 0.66A - .002 = Hight of Side

D = .173A + .015

 $E = \frac{1}{2}F = \text{Depth of Slot}$

F = .134B + C = Hight of Head.



A	. В	С	D	E	F
.060	.0894	.0376	.025	.025	.0496
.073	.1107	.0461	.028	.030	.0600
.08 6	.132	.0548	.030	.036	.0725
.099	.153	.0633	.032	.042	.0838
.112	.1747	.0719	.034	.048	.0953
.125	.196	.0805	.037	.053	.1068
.138	.217	.089	.039	.059	.1180
.151	.2386	.0976	.041	.065	.1296
.164	.2599	.1062	.043	.071	.1410
.177	.2813	.1148	.046	.076	.1524
.190	.3026	.1234	.048	.082	.1639
.216	•3452	.1405	.052	.093	.1868
.242	.3879	.1577	.057	.105	.2097
.268	.4305	.1748	.061	.116	.2325
•294	.4731	.192	.066	.128	•2554
.320	.5158	.2002	.070	.140	.2783
.346	.5584	.2263	.075	.150	3011
•372	.601	.2435	.079	.162	.3240
.398	.6437	.2606	.084	.173	.3469
.424	.6863	.2778	.088	.185	.3698
.450	.727	.295	.093	.201	.4024

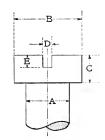
FLAT FILLISTER HEAD SCREWS

A = Diameter of Body

B = 1.64A - .009 = Diam. of Head.

C = 0.66A - .002 = Hight of HeadD = 0.173A + .015 = Width of Slot

 $E = \frac{1}{2}C = \text{Depth of Slot}$



A	В	С	D	E
.060 .073 .086 .099	.0894 .1107 .132 .153	.0376 .0461 .0548 .0633	.025 .028 .030 .032	.019 .023 .027 .032 .036
.125 .138 .151 .164	.196 .217 .2386 .2599 .2813	.0805 .0890 .0976 .1062	.037 .039 .041 .043	.040 .044 .049 .053
.190 .216 .242 .268	.3026 .3452 .3879 .4305 .4731	.1234 .1405 .1577 .1748	.048 .052 .057 .061	.062 .070 .079 .087
.320 .346 .372 .398 .424	.5158 .5584 .601 .6437 .6863	.2092 .2263 .2435 .2606 .2778	.070 .075 .079 .084 .088	.104 .113 .122 .130
.450	.727	.295	.093	.147

FLAT HEAD MACHINE SCREWS. A. S. M. E. STANDARD

FLAT HEAD SCREWS

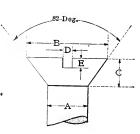
A = Diameter of Body

B = 2A - .008 = Diam. of Head

 $C = \frac{A - .008}{1.739} = Depth of Head$

D = .173A + .015 = Width of Slot*

 $E = \frac{1}{3}C = Depth of Slot$



A	В	С	D	E
.060 .073 .086 .099 .112 .125 .138	.112 .138 .164 .190 .216	.029 .037 .045 .052 .060 .067 .075	.025 .028 .030 .032 .034	.010 .012 .015 .017 .020
.164 .177	.320 .346	.090 .097	.043 .046	.030 .032
.190 .216 .242 .268 .294	•37 ² •4 ² 4 •47 ² •5 ² 8	.105 .120 .135 .150	.048 .052 .057 .061 .066	.035 .040 .045 .050
.320 .346 .372 .398 .424	.632 .682 :732 :788 .840	.179 .194 .209 .224 .239	.070 .075 .079 .084 .088	.060 .065 .070 .075 .080
-4 50	.892	•254	.093	.085

ROUND HEAD MACHINE SCREWS. A. S. M. E. STANDARD

ROUND HEAD SCREWS

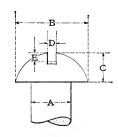
A = Diameter of Body

B = 1.85A - .005 = Diam. of Head

C = .7A = Hight of Head

D = .173A + .015 =Width of Slot

 $E = \frac{1}{2}C + .oi = Depth of Slot$



A	В	С	D	Е
.060 .073 .086 .099 .112 .125 .138 .151 .164 .177 .190 .216 .242 .268	.106 .130 .154 .178 .202 .226 .250 .274 .298 .322 .346 .394 .443 .491	.042 .051 .060 .069 .078 .087 .096 .105 .114 .123 .133 .151 .169 .187	.025 .028 .030 .032 .034 .037 .039 .041 .043 .046	.031 .035 .040 .044 .049 .053 .058 .062 .067 .071 .076 .085 .094 .112
•320 •346 •372 •398 •424	.587 .635 .683 .731 .779	.224 .242 .260 .278 .296	.070 .075 .079 .084 .088	.122 .131 .140 .149 .158







HOT PRESSED AND COLD PUNCHED NUTS

U. S. S ₁	randard Cold Pui	Hot Prinched N	ESSED AND UTS	COLD P	UNCHED CI	HECK AND	Jam Nuts
H	IEXAGON	AND SQU	ARE		HEX	AGON	
Dia. Bolt	Across Flats	Thick- ness	Dia. Hole	Dia. Bolt	Across Flats	Thick- ness	Dia. Hole
14 5 6 7 6 9 6 5 6 5 6 6 6 6 6 6 6 6 6 6 6 6 6 6	1218710552 9217212378	*(4*5 H-03 0*7 F0 0;H-05 000 41- 0	8 6 9,141-623052 114-161-631-3352	$\begin{array}{c} \frac{1}{4} \\ \frac{5}{16} \\ \frac{3}{38} \\ \frac{7}{16} \\ \frac{1}{2} \end{array}$	101-101-100-010 101-101-1-10-10-10	$\begin{array}{c} \frac{3}{16} \\ 7 \\ 3 \\ 2 \\ 1 \\ 4 \\ \hline 15 \\ \hline 5 \\ \hline 16 \\ \end{array}$	16 14 9 41 (21 8)24 16 10 10 10 10 10 10 10 10 10 10 10 10 10
9 16 5 8 34 4 7 8	$\begin{array}{c} \frac{3}{3}\frac{1}{2} \\ 1\frac{1}{16} \\ 1\frac{1}{4} \\ 1\frac{7}{16} \\ 1\frac{5}{8} \end{array}$	9 15 5 8 3 4 7 8	হালজাল নুন্দান্ত থালজালচা∞ব¦লথাজ	9 15 5 8 3 4 7 8 1	$ \begin{array}{c} \frac{3}{3}\frac{1}{2} \\ 1\frac{1}{16} \\ 1\frac{1}{4} \\ 1\frac{7}{16} \\ 1\frac{5}{8} \end{array} $	11/32/38 7/16 12/9	9,40,14 1,47,0 160,00 150 416 210
1 13 1 13 1 13 1 13 1 13 1 13 1 13 1 13	$ \begin{array}{c} 1\frac{1}{16} \\ 2 \\ 2\frac{3}{16} \\ 2\frac{3}{8} \\ 2\frac{9}{16} \end{array} $	I 1/8 1/4 2/8 1/2 1/2 1/2 1/8 1/2 1/8 1/2 1/8 1/8 1/8 1/8 1/8 1/8 1/8 1/8 1/8 1/8	1.5 1.6 1.5 1.5 1.5 2.9 2.25 1.64	1 \frac{1}{8} 1 \frac{1}{4} 1 \frac{3}{8} 1 \frac{1}{2} 1 \frac{5}{8}	$ \begin{array}{c c} 1\frac{13}{16} \\ 2 \\ 2\frac{3}{16} \\ 2\frac{3}{8} \\ 2\frac{9}{16} \end{array} $	5/82/44/17/81/17/81/17	$ \begin{array}{c} 156 \\ 116 \\ 132 \\ 132 \\ 132 \\ 132 \\ 134 $
$1\frac{3}{4}$ $1\frac{7}{8}$ 2 $2\frac{1}{8}$ $2\frac{1}{4}$	$\begin{array}{c} 2\frac{3}{4} \\ 2\frac{1}{1}\frac{5}{6} \\ 3\frac{8}{1}\frac{5}{6} \\ 3\frac{1}{2} \end{array}$	$1\frac{3}{4}$ $1\frac{7}{8}$ 2 $2\frac{1}{8}$ $2\frac{1}{4}$	1258 2823505656 12256 2823505656 11116	1 \frac{3}{4} 1 \frac{7}{8} 2	$ \begin{array}{c} 2\frac{3}{4} \\ 2\frac{1}{16} \\ 3\frac{1}{8} \end{array} $	I I 16 I 8	I 1255 I 1250 I 232
$2\frac{3}{8}$ $2\frac{1}{2}$ $2\frac{3}{4}$ 3	$ \begin{array}{c} 3\frac{11}{16} \\ 3\frac{78}{8} \\ 4\frac{1}{4} \\ 4\frac{5}{8} \end{array} $	23812334 2234 3	$\begin{array}{c} 2\frac{1}{16} \\ 2\frac{1}{6}\frac{1}{2} \\ 2\frac{1}{6}\frac{1}{2} \\ 2\frac{1}{6}\frac{1}{2} \\ 2\frac{1}{8} \\ 2\frac{1}{8} \end{array}$				

Finished case-hardened and semi-finished nuts are made to the above dimensions. Semi-finished nuts are tapped and faced true on the bottom.





HOT PRESSED NUTS

HOT PRESSED NUTS, MANUFACTURERS HOT PRESSED AND FORGED NUTS, MANUFACTURERS STANDARD

	HE	XAGON			SQ	UARE	
Dia. Bolt	Across Flats	Thick- ness	Dia. Hole	Dia. Bolt	Across Flats	Thick- ness	Dia. Hole
14566 13876	1225/833/47-18 I	14 56 38 7 16 12	7 30 31 10 246 7 6	14 5 6 28 7 16 12	1/215/k02/417/8 I	14 55 16 38 716 12	7 32 22 25 44 6 7 6 T 6
9 15 15 18 3 14 7 8 1	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	9.66 1.588344758	102 9 to 1/01/01 1 2/00/01/01	9 6 5 8 8 8 4 7 18 1	$ \begin{array}{c c} & 1\frac{1}{8} \\ & 1\frac{1}{4} \\ & 1\frac{1}{2} \\ & 1\frac{3}{4} \\ & 2 \end{array} $	9 15 8 8 24 7 8	152 0 H 240021007-10
1 1 4 3 8 1 1 2 5 8 1 2 5 8	2 21 21 21 23 23 3	14381 12581 125834	$\begin{array}{c} \frac{3}{3}\frac{1}{2} \\ I \frac{3}{3}\frac{1}{2} \\ I \frac{3}{16} \\ I \frac{5}{16} \\ I \frac{7}{16} \\ \end{array}$	1 1/8 1 1/4 1 1/2 1 1/2 1 1/2 1 1/8	$ \begin{array}{c} 2\frac{1}{4} \\ 2\frac{1}{2} \\ 2\frac{3}{4} \end{array} $ $ 3\frac{1}{4} $	1 1/8 1 1/4 1 1/2 1 1/2 1 1/2 1 1/2	$\begin{array}{c} \frac{3}{3}\frac{1}{2} \\ \mathbf{I}\frac{3}{3}\frac{2}{2} \\ \mathbf{I}\frac{3}{3}\frac{5}{2} \\ \mathbf{I}\frac{5}{16} \\ \mathbf{I}\frac{5}{16} \\ \mathbf{I}\frac{7}{16} \end{array}$
$1\frac{3}{4}$ $1\frac{7}{8}$ 2 $2\frac{1}{8}$ $2\frac{1}{4}$	3 3 3 3 3 3 3 3	$1\frac{7}{8}$ 2 2 $2\frac{1}{8}$ $2\frac{1}{4}$	1 9 6 1 1 1 1 6 1 1 1 1 6 1 7 8 1 7 8 2	$ \begin{array}{c} 1\frac{3}{4} \\ 1\frac{7}{8} \\ 2 \\ 2\frac{1}{4} \end{array} $	3 ¹ / ₂ 3 ³ / ₄ 4 4 4 4 ¹ / ₄	134 178 2 218 24	$ \begin{array}{c c} I & \frac{9}{16} \\ I & \frac{1}{16} \\ I & \frac{1}{16} \\ I & \frac{1}{16} \\ I & \frac{1}{8} \end{array} $

 $2\frac{3}{8}$ $2\frac{1}{2}$ $2\frac{3}{4}$

 $\begin{array}{c} 2\frac{1}{8} \\ 2\frac{1}{4} \\ 2\frac{7}{16} \\ 2\frac{11}{16} \\ 2\frac{1}{16} \\ 3\frac{1}{8} \end{array}$

 $2\frac{3}{8}$ $2\frac{1}{2}$ $2\frac{3}{4}$

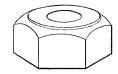
3

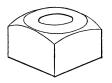
 $\tilde{3}^{\frac{1}{4}}$ $3\frac{1}{2}$

5 5 5

 $4\frac{1}{4}$ $4\frac{1}{2}$ $4\frac{3}{4}$ 5 $5\frac{1}{2}$

 $\begin{array}{c} 2\frac{1}{8} \\ 2\frac{1}{4} \\ 2\frac{7}{16} \\ 2\frac{1}{16} \\ 2\frac{1}{16} \\ 3\frac{1}{8} \end{array}$





COLD PUNCHED NUTS, MANUFACTURERS STANDARD

	Не	KAGON			5	SQUARE	
Dia. Bolt	Across Flats	Thick- ness	Dia. Hole	Dia. Bolt	Across Flats	Thick- ness	Dia. Hole
14 Tropo 7 Tropo 12 12 12 12 15 Tropo 14 Tropo 15 Tropo 1	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 2 2 2 3 3 3 3	145 F000 7 T1 62 152 6 F0	To performed to the to to to the former describes described and the total to the following sectors of the following secto	14 "Trops Trops 15 15 15 15 15 15 15 15 15 15 15 15 15	1216623416278 I 121684381671F116686464 2 141447634 1417164 3 3 3 3 3 3 4	14 1 1 1 1 1 1 1 1 2 1 1 2 1 1 2 1 1 2 1	7.3.0.73.1.(3.0.15.7.16.7.6.7.6.7.16.10.16.16.16.16.7.7.7.6.16.16.16.16.16.16.16.16.16.16.16.16.1



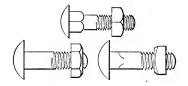


HOT PRESSED NUTS. MANUFACTURERS STANDARD NARROW GAGE SIZES

	Hex	AGON		. Square				
Dia. Bolt	Across Flats	Thick- ness	Dia. Hole	Dia. Bolt	Across Flats	Thick- ness	Dia. Hole	
3677112116 10000415 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	1000 100 100 100 100 100 100 100 100 10	TEST	2 6 4 7 6 6 7 7 6 7 7 7 6 6 7 7 7 6 6 7 7 7 6 6 7 7 7 6 7 7 7 6 7 7 7 7 6 7	1(6 7) 44 C 16 38 S 7 (1 42 7) 5 8 S 34 4 7 (8 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	10000000000000000000000000000000000000	COLUMN CO	0 0 0 7 12 12 12 14 14 14 17 17 18 14 12 12 12 12 12 12 12 12 12 12 12 12 12	

DIMENSIONS OF WHITWORTH STANDARD HEXAGONAL NUTS AND BOLT-HEADS

Dia. of Bolt	Width of Nut or Bolt Head across Flats	Hight of Bolt Head	Dia. of Bolt	Width of Nut or Bolt Head across Flats	Hight of Bolt Head
1 (6 2 1) 1 4 6 6 6 6 6 7 7 1 1 2 7 1 6 6 8 1 1 6 3 4 2 7 7 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	.338 .448 .525 .601 .709 .820 .919 I.011 I.101 I.301 I.390 I.479 I.574 I.670 I.860	.109 .164 .219 .273 .328 .383 .437 .492 .547 .601 .656 .711 .766 .820 .875	I	2.048 2.215 2.413 2.576 2.758 3.018 3.149 3.337 3.546 3.750 3.894 4.049 4.181 4.346 4.531	1.094 1.203 1.312 1.422 1.531 1.641 1.750 1.859 1.960 2.078 2.187 2.297 2.406 2.516 2.625



BUTTON HEAD MACHINE AND LOOM BOLTS HOOFES & TOWNSEND CO.

Diameter Bolt Diameter Head. Thickness of Head	$\begin{array}{c} \frac{3}{16} \\ \frac{1}{2} \\ \frac{5}{32} \end{array}$	1 5 8	5 16 13 16	$\frac{\frac{3}{8}}{\frac{15}{16}}$	$1\frac{\frac{7}{16}}{16}$ $\frac{1}{4}$	$1\frac{1}{2}$ $1\frac{1}{8}$ $\frac{1}{4}$	$1\frac{\frac{9}{16}}{16}$	1 8 3 8	3 4 15 18	1 8 5 8	$ \begin{array}{c c} I \\ 2\frac{1}{8} \\ \hline \frac{3}{4} \end{array} $
	3 2		0-	10	*	*	1.0		-	•	*

CARRIAGE BOLTS UPSON NUT CO.

$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	$\begin{array}{c c} \frac{1}{6} & 1\frac{1}{16} & 2 \\ & \frac{7}{16} & \end{array}$	$\frac{2\frac{1}{16}}{\frac{1}{2}}$
---	--	-------------------------------------

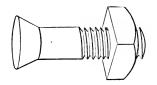
LENGTHS OF BOLTS

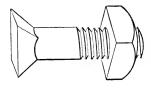
SQUARE HEAD, Hexagon Head, Button Head, Round Head and Cone Head bolts are measured under the head. Countersunk Head bolts, bolt ends and rods are measured over all.

LENGTHS OF THREADS CUT ON BOLTS

Length of Bolts	1 & 15	3 & 17 g	½ & 18	5 8	34	7 8	1	1 1/8	114
I to $1\frac{1}{2}''$ $1\frac{5}{8}$ to $2\frac{1}{2}''$ $2\frac{5}{9}$ to $2\frac{1}{2}''$ $2\frac{5}{9}$ to $3\frac{1}{2}''$ $3\frac{1}{8}$ to $4\frac{1}{2}''$ $4\frac{1}{8}$ to $8\frac{1}{2}''$ $12\frac{1}{8}$ to $20\frac{1}{2}''$	3 4 7 8 1 1 1 1 1	78 I I I 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	$\begin{array}{c} I \\ I \\ I \\ I \\ I \\ \frac{1}{4} \\ I \\ \frac{1}{2} \\ 2 \\ \end{array}$	$\begin{array}{c} \mathbf{I} \frac{1}{4} \\ \mathbf{I} \frac{1}{4} \\ \mathbf{I} \frac{1}{4} \\ \mathbf{I} \frac{1}{4} \\ \mathbf{I} \frac{1}{2} \\ \mathbf{I} \frac{1}{2} \\ \mathbf{I} \frac{1}{2} \\ \mathbf{I} \frac{3}{4} \\ 2 \end{array}$	$\begin{array}{c} \cdot \cdot \\ I\frac{1}{2} \\ I\frac{1}{2} \\ I\frac{1}{2} \\ I\frac{1}{2} \\ I\frac{3}{4} \\ 2 \\ 2 \end{array}$	$\begin{array}{c} \cdot \cdot \cdot \\ I\frac{3}{4}\\I\frac{3}{4}\\I\frac{3}{4}\\I\frac{3}{4}\\I\frac{3}{4}\\2\frac{1}{2}\\2\frac{1}{2}\end{array}$	$ \begin{array}{c} $	2 1/4 2 1/2 2 3/4 3 1/4 3 1/4	2½ 3 3½ 3½ 3½

Bolts longer than 20 inches and larger than $1\frac{1}{4}$ inch in diameter are usually threaded about 3 times the diameter of the rod.



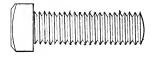


ROUND AND SQUARE COUNTERSUNK HEAD BOLTS

Diameter Bolt Diameter Round Head	1 1 2	$\frac{5}{16}$ $\frac{1}{3}$ $\frac{9}{2}$	$\frac{3}{8}$ $\frac{11}{16}$	$\frac{7}{16}$ $\frac{25}{32}$	1278	$\frac{9}{\frac{16}{3}\frac{3}{1}}$	$1\frac{\frac{5}{8}}{1\frac{1}{6}}$	$\frac{3}{4}$ $1\frac{1}{4}$	$1\frac{\frac{7}{8}}{1\frac{7}{16}}$	I 1 5/8
Distance across Flats Square Head Thickness Square and							1 1 6	11/4	1 7 1 6	1 5/8
Round Heads	3 16	$\frac{3}{16}$	$\frac{7}{32}$	1/4	1/4	$\frac{9}{32}$	5 16	$\frac{1}{3}\frac{1}{2}$	$\frac{1}{3}\frac{3}{2}$	7 16

TAP BOLTS





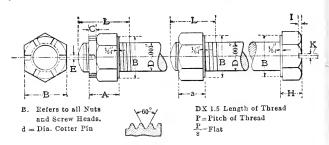
Diameter Bolt No. of Threads per Inch	20	16 18	3 16	7 16 14	$\frac{1}{2}$	$\frac{9}{16}$	5 11	3 4 10	7 8 9	8
Across Flats Hex. and Square Heads	3 8 7 16 17	$\frac{15}{32}$ $\frac{9}{16}$ $\frac{21}{32}$		212 334 156 38	$\frac{\frac{3}{4}}{\frac{5}{6}}$ $\frac{1}{16}$ $\frac{7}{16}$	$\begin{array}{c} \frac{27}{32} \\ \frac{31}{32} \\ \frac{31}{2} \\ \frac{3}{16} \\ \end{array}$	$ \begin{array}{r} $	$ \begin{array}{c} 1_{8}^{1} \\ 1_{6}^{19} \\ 1_{3}^{19} \\ 1_{3}^{19} \\ 2 \end{array} $	1 5 6 8 4 5 4 1 5 6 4 3 4 4	$ \begin{array}{c} \mathbf{I}\frac{1}{2} \\ \mathbf{I}\frac{2}{3} \\ \mathbf{I}\frac{2}{3} \\ 2\frac{1}{8} \end{array} $

STOVE BOLT DIAMETERS AND THREADS





Dia. of Bolt	18	5 32	3 16	7 3 2	14	5 16	38
No. of Threads per Inch							



All heads and nuts to be semi-finished. All screws to be of steel not less than 100,000 pounds tensile strength and 60,000 pounds elastic limit per square inch. Screws, screw heads and plain nuts to be left soft. Castle nuts to be case-hardened. Where screws are to be used in soft material such as cast iron, brass, bronze or aluminum, the U. S. S. pitches are to be used. Body diameter to be 0.001 inch less than nominal diameter with a plus tolerance of zero and a minus tolerance of 0.002 inch. Nuts shall fit without perceptible shake. This was originally known as the A. L. A. M. Standard, but is now the S. A. E. (Society Automobile Engineers).

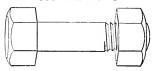
AUTOMOBILE SCREW AND NUT STANDARDS ADOPTED BY THE S. A. E.

D	1/4	$\frac{5}{16}$	38	<u>-7</u> €	$\frac{1}{2}$	T 6	<u>5</u> 8	$\frac{1}{1}\frac{1}{6}$	34	78	I
P	28	24	24	20	20	18	18	16	16	14	14
A	$\frac{9}{32}$	$\frac{2}{6}\frac{1}{4}$	$\frac{1}{3}\frac{3}{2}$	2 9 6 4	$\frac{9}{16}$	$\frac{3}{6}\frac{9}{4}$	$\frac{2}{3}\frac{3}{2}$	49 64	$\frac{1}{1}\frac{3}{6}$	$\frac{29}{32}$	I
a	7 3 2	$\frac{1}{6}\frac{7}{4}$	21 64	38	7 16	$\frac{3}{6}\frac{1}{4}$	$\frac{3}{6}\frac{5}{4}$	$\frac{1}{3}\frac{9}{2}$	$\frac{\overline{2}}{3}\frac{1}{2}$	$\frac{49}{64}$	78
В	7 16	$\frac{1}{2}$	$\frac{9}{16}$	<u>5</u> 8	34	78	$\frac{15}{16}$	I	1 1 6	I 1/4	$1\frac{7}{16}$
С	3 2	$\frac{3}{32}$	1/8	18	3 16	$\frac{3}{16}$	1/4	1/4	1/4	1/4	$\frac{1}{4}$
Е	5 64	5 64	18	18	1/8	$\frac{5}{32}$	3 2	$\frac{5}{82}$	$\frac{5}{32}$	$\frac{5}{32}$	$\frac{5}{32}$
Н	3 T 6	$\frac{1}{6}\frac{5}{4}$	9 32	$\frac{2}{6}\frac{1}{4}$	38	$\frac{27}{64}$	$\frac{1}{3}\frac{5}{2}$	$\frac{3}{6}\frac{3}{4}$	9 16	$\frac{2}{3}\frac{1}{2}$	$\frac{3}{4}$
Ι	$\frac{3}{32}$	674	1 8	18	1/8	1/8	1/8	1/8	1/8	1/8	1/8
K	$\frac{1}{16}$	1 6	$\frac{3}{32}$	$\frac{3}{32}$	$\frac{3}{32}$	$\frac{3}{3}\vec{2}$	$\frac{3}{32}$	3 2	$\frac{3}{32}$	3 3 2	$\frac{3}{32}$
d	$\frac{1}{16}$	$\frac{1}{16}$	3 2	$\frac{3}{32}$	3 2	1/8	1/8	18	1/8	1 8	18
L	3/8	$\frac{15}{32}$	9 16	721 32	34	27 32	15 16	$1\frac{1}{32}$	1 ½	1 5 6	I 1/2

PLANER NUTS

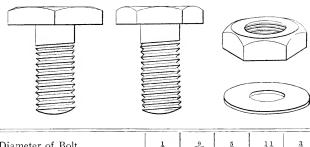
1

COUPLING BOLTS



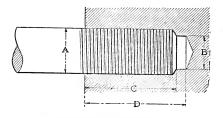
	1		
Diameter of Bolt	$\frac{1}{2}$ $\frac{5}{8}$	$\frac{3}{4}$ $\frac{7}{8}$	I 11 11 11
No. of Threads per Inch	13 11	10 9	8 7 7
Short Diameter of Head	7 116	$1\frac{1}{4} 1\frac{7}{16}$	$ 1\frac{5}{8} 1\frac{1}{16} 2$
Length of Head	$\frac{1}{2}$ $\frac{5}{8}$	$\frac{3}{4}$ $\frac{7}{8}$	I I 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1
Thickness of Nut	1 5	$\frac{3}{4}$ $\frac{7}{8}$	$I = I \frac{1}{8} = I \frac{1}{4}$
Length of Head Thickness of Nut Short Diameter of Nut	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	$1\frac{1}{4} 1\frac{7}{18}$	$1\frac{5}{8}$ $1\frac{13}{16}$ 2
	" 1"	1 10	" "

PLANER HEAD BOLTS AND NUTS



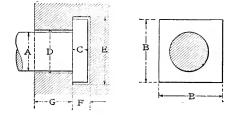
Diameter of Bolt		$\frac{9}{16}$	<u>5</u>	$\frac{1}{1}\frac{1}{6}$	34
No. of Threads per Inch	12	12	12	12	12
Short Diameter of Head		I	I 1/8	1 ½	$1\frac{1}{4}$
Length of Head	-5 16	1 5 1 6	38	3 8	7 16
Short Diameter of Nuts	$1\frac{1}{8}$	18	11	11/4	17
Thickness of Nuts	3 8	3 8	38	3 8	1/2
Washers for Planer Head Bolts.	1/2	16	5 8	11	3
Diameter of Washers	176	175	1 2	I 18	118
Thickness of Washers	18	16	1	10	3
	10	10	10	10	32

DEPTHS TO DRILL AND TAP FOR STUDS

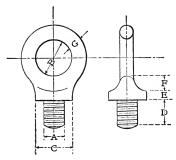


Dia. of Stud Dia. of Drill C. I Depth of Thread Depth to Drill	A B C D	$\begin{array}{c} \frac{1}{4} \\ \frac{1}{3} \\ \frac{3}{6} \\ \frac{3}{8} \\ \frac{7}{16} \end{array}$	$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	$\begin{array}{c} 7 \\ \hline 16 \\ \hline 38 \\ \hline 212 \\ \hline 232 \\ \hline 32 \\ \hline 32 \\ \hline \end{array}$	1 2 2 6 3 4 2 7 2 3 2 2 3 2 2 2 3 2 2 2 2 3 2 3 2 3	9 1361+7-2156	$ \begin{array}{r} $	34 414 18 14	$\begin{bmatrix} \frac{7}{8} \\ \frac{3}{4} \\ \frac{1}{16} \\ \frac{7}{16} \end{bmatrix}$	1 5 5 4 1 12 1 58
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BOLT HEADS FOR STANDARD T-SLOTS



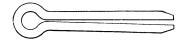
Dia. of Bolt	Width of Head	Thickness of Head	Width of Slot	Width of Slot	Depth of Slot	Maximum Depth with St'd Cutter
A	В	С	D	Е	F	G
14.57 3380 7 6 24555 334 718	1.558.24772.188.149.568.11116	100 s/6 s/6 s/14 cs/00 + (215)00 cs/4	5 6 000 7 6 12 15 00 00 4 7 10 I	5.8 166 x 665 66 x 665 66 x 14.5 16.5 16 x 14.5 16.5 16 x 14.5 16.5 16 x 14.5 16.5 16 x 15.5 16	5 62 7 22 9 22 8 22 7 22 1 6 5 16 5 2 7 2 1 16 5 16 11 11 11 11 11 11 11 11 11 11 11 11	367 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7 7



EYE BOLTS

A	В	С	D	E	F	G
200 - 170 - 100 - 170 -	2 2 2 2 2 2 3 3 3 3 3 3 3 3 3	214 1417 5 40 7 5 1 4	10 1/21/4-10/2-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1-1	or 16 − 4 ± 5 16 ° 5 10 ≈ 20 7 10 − 62 ± 12 ± 16 ± 50 € 1 − 62 ± 10 € 10 − 40 .	360-120.5[6] 1 0 0 0 0 1 1 1 1 1 1 1 1 1 1 1 1 1 1	5 6 8 1 2 9 6 6 6 8 4 5 6 6 8 1 1 1 1 1 4 6 6 8 1 1 1 1 1 4 6 6 8 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1

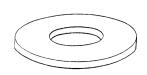
SPRING COTTERS



No. of Wire Gage	13 32 ½ to 2	12 ⁷ 64 ½ to 2	11 1/8 1/2 to 21/2	10 9 64 ½ to 2½	9 32 ½ to 2½	8 11 64 ½ to 2½	7 3 16 3 to 3	6 13 64 3 to 3
No. of Wire Gage Dia. Inches Lengths. Inches	5 32 1 to 3	4 1 1 to 4	I 5 16 1 to 4	3 12 to 4	76 18 to 5	1 2 to 6	3 to 6	

Regular Lengths vary by $\frac{1}{4}$ inch up to 4 inches and by 1 inch from 4 to 6 inches. Lengths are measured under the eye.

ROUND AND SQUARE WASHERS



U. S. STANDARD WASHERS

Outside

Diam-

Size of Hole

Size of Bolt Thickness

Wire Gage

Don	11010	eter	No.	
			-	
$\frac{3}{16}$	1/4	9 16	$18 \left(\frac{3}{64} \right)$	
$\frac{1}{4}$	$\frac{5}{16}$	34	$16 \left(\frac{1}{16} \right)$	
$\frac{5}{16}$	3 8	$\frac{7}{8}$	$16 \left(\frac{1}{16} \right)$	=
38	7 16	1	$14 \left(\frac{5}{64} \right)$	
$\frac{7}{16}$	$\frac{1}{2}$	I 1/4	$14 \left(\frac{5}{64} \right)$	
$\frac{1}{2}$	$\frac{9}{16}$	1 3	$12 \left(\frac{3}{32} \right)$	
$\frac{9}{16}$	5 8	$1\frac{1}{2}$	$12 \left(\frac{3}{32}\right)$	
<u>5</u> 8	$\frac{11}{16}$	$1\frac{3}{4}$	$10(\frac{1}{8})$	-
$\frac{3}{4}$	13 16	2	$10 \left(\frac{1}{8}\right)$	١,
$\frac{7}{8}$	15 16	$2\frac{1}{4}$	$9 \left(\frac{5}{32} \right)$	'
1	$1\frac{1}{16}$	$2\frac{1}{2}$	$9 \left(\frac{5}{32} \right)$	_
I 1/8	I 1/4	$2\frac{3}{4}$	$9(\frac{5}{32})$	1
$I\frac{1}{4}$	1 3/8	3	$9 \left(\frac{5}{32} \right)$	
$1\frac{3}{8}$	I 1/2	31/4	$8 \left(\frac{1}{6} \frac{1}{4} \right)$	
$1\frac{1}{2}$	1 5	$3\frac{1}{2}$	$8 \left(\frac{11}{64} \right)$	
1 5	$1\frac{3}{4}$	334	$8 \left(\frac{1}{6} \frac{1}{4} \right)$	
$1\frac{3}{4}$	I 7/8	4	$8 \left(\frac{1}{6} \frac{1}{4} \right)$	
$1\frac{7}{8}$	2	$4\frac{1}{4}$	$8 \left(\frac{1}{6} \frac{1}{4} \right)$	
2	21/8	$4\frac{1}{2}$	$8 \left(\frac{11}{64} \right)$	
$2\frac{1}{4}$	28	$4\frac{3}{4}$	$6 \left(\frac{3}{16} \right)$	
2 ½	$2\frac{5}{8}$	5	$5 \left(\frac{7}{32} \right)$	
				-

NARROW GAGE WASHERS

Size of Bolt	Size of Hole	Outside Diam- eter	Thickness Wire Gage No.						
14500 76 225 Possoderio 140 1400 121 1	5 1 0 0 1 0 1 0 0 1 0 0 0 0 0 0 0 0 0 0	558324788451111111111111111111111111111111111	$\begin{array}{c} 16 \left(\frac{1}{16}\right) \\ 16 \left(\frac{1}{16}\right) \\ 16 \left(\frac{1}{16}\right) \\ 16 \left(\frac{1}{16}\right) \\ 14 \left(\frac{3}{647}\right) \\ 12 \left(\frac{3}{32}\right) \\ 12 \left(\frac{3}{32}\right) \\ 10 \left(\frac{1}{8}\right) \\ 10 \left(\frac{1}{8}\right) \\ 9 \left(\frac{5}{32}\right) \\ 9 \left(\frac{5}{32}\right) \\ 9 \left(\frac{5}{32}\right) \\ 9 \left(\frac{5}{32}\right) \\ 8 \left(\frac{1}{64}\right) \\ 8 \left(\frac{1}{64}\right) \end{array}$						

SQUARE WASHERS STANDARD SIZES

Size of Bolt	Size of Hole	Width	Thickness
23/67 10 10/62/47/8 I 1/61/42/80 12/20/4	7 1 1 (2) 1 (2) (5) (6 (2) (2) (1) (2) (5) (6 (2) (2) (2) (2) (2) (2) (2) (2) (2) (2)	1 ½ 1 ½ 2 2 ¼ 4 2 ½ 3 3 ½ 4 4 ½ 5 5 6 6 6 6	1/2 - 12 ⁻¹³ 1 - 14 - 14 - 14 - 14 - 14 - 14 - 15 - 15



Cast-Iron Washers

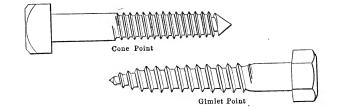
Size of Bolt	Outside Diameter	Thickness	Size of Bolt	Outside Diameter	Thickness
200 1(215)802(417)0 H	$ \begin{array}{c} 1\frac{1}{2} \\ 2 \\ 2\frac{1}{2} \\ 3\frac{1}{3} \\ 4 \end{array} $	5 <mark>,1</mark> 6,016 ±(\$15,16,01±1-10	I 1/8 I 1/4 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	$ \begin{array}{c} 4\frac{1}{2} \\ 5 \\ 5\frac{1}{2} \\ 6 \\ 7 \\ 7\frac{1}{2} \end{array} $	I I 168 I 143388 I I 15458 I I 158

RIVETING WASHERS

Size of Rivet	Size of Hole	Outside Diam- eter	Thick- ness Wire Gage	Size of Rivet	Size of Hole	Outside Diameter	Thickness Wire Gag
7 (.180) 6 (.203) 5 (.220) $\frac{1}{4}$ $\frac{1}{4}$ $\frac{1}{16}$ $\frac{1}{16}$ $\frac{1}{38}$ $\frac{1}{38}$	3 1	1619,66 115,505,503,43,447,527,88 I	18 18 18 16 16 14 14 14	3)87 67 66 11 7 67 16 16 16 16 16 16 16 16 16 16 16 16 16	03 2315 6316 6317- 6315-	1 4 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	12 14 12 12 12 11 11

MACHINE AND WOOD SCREW GAGE

No. of Screw Gage	Size of Number in Decimals	No. of Screw Gage	Size of Number in Decimals	No. of Screw Gage	Size of Number in Decimals	No. of Screw Gage	Size of Number in Decimals
000	.03152	12	.21576	25	.38684	38	-55792
00	.04468	13	.22802	26	.40000	39	.57108
0	.05784	14	.24208	27	.41316	40	.58424
1	.07100	15	25524	28	.42632	41	-59740
2	.08416	16	.26840	29	.43948	42	.61056
3	.00732	17	.28156	30	.45264	43	.62372
4	.11048	18	.29472	31	.46580	44	.63688
5	.12364	19	.30788	32	.47896	45	.65004
ŏ	.13680	20	.32104	33	.49212	46	.66320
7	.14996	21	-33420	34	.50528	47	.67636
7 8	.16312	22	.34736	35	.51844	48	.68952
9	.17628	23	.36052	36	.53160	49	.70268
10	.18944	24	.37368	37	54476	50	.71584
	.20260	Th	e difference	between co	nsecutive si	zes is .o1	316"



COACH AND LAG SCREWS

			l I	I		1			i	
Diameter Screw	1/4	5 I 6	38	$\frac{7}{16}$	$\frac{1}{2}$	$\frac{9}{16}$	<u>5</u>	34	7 8	I
No. of Threads per Inch	10	$9\frac{1}{2}$	7	7	6	5	5	$4\frac{1}{2}$	$4\frac{1}{2}$	3
Across Flats Hex. and										
Square Heads		$\frac{1}{3}\frac{5}{2}$	9 16	$\frac{21}{32}$	34	$\frac{27}{32}$	15	18	$1\frac{5}{16}$	$1\frac{1}{2}$
Thickness Hex. and Square				1						
Heads	$\frac{3}{16}$	1/4	15 16	3 8	$\frac{7}{16}$	$\frac{1}{2}$	$\frac{1}{3}\frac{7}{2}$	<u>5</u>	$\frac{3}{4}$	78
•	l				ļ		1	t		1

Lengths of Threads on Coach and Lag Screws Of all diameters

Length of Screw	Length of Thread	Length of Screw	Length of Thread
1½" 2½" 2½" 3½" 3½" 4½"	To Head 1½" 2 " 2¼" 2½" 3 " 3½"	5" 52" 6" 7" 8" 9"	4 " 4 1" 4 2" 5 " 6 " 7 "

LAG-SCREW TEST (SCREWS DRAWN OUT OF YELLOW PINE)

Test by Hoopes and Townsend

Diameter Screw	 $\frac{1}{2}$ in.	5 in.	3 in.	7 in.	ı in.
Depth in Wood	 $3\frac{1}{2}$ in.	4 in.	4 in.	5 in.	6 in.
Force in Pounds	 4,960	6,000	7,685	11,500	12,620

WOOD SCREWS

Wood screws range in size from No. o to No. 30, by the American Screw Company's gage and in lengths from $\frac{1}{4}$ inch to 6 inches. The increase in length is by eights of an inch up to 1 inch, then by quarters of an inch up to 3 inches and by half inches up to 5 inches. As a rule the threaded portion is about seven tenths of the total length. The included angle of the flat head is 82 degrees. The table below gives the body and head diameters, and the threads per inch as generally cut, although there is no fixed standard as to number of threads which is universally adhered to by all wood-screw manufacturers.

Flat headed wood-screws include the head in the length given. With round headed screws, one half the head is generally included

in the length although the practice is not uniform.



Wood-Screw Dimensions

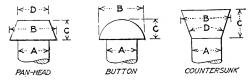
(ANGLE OF FLAT HEAD = 82 DEGREES)

						·	
No. of Screw Gage	Diameter of Screw	Diameter of Head	Threads per Inch	No. of Screw Gage	Diameter of Screw	Diameter of Head	Threads per Inch
0 1 2 3 4 4 5 6 7 8 9 10 11 12 13 14 15	$\begin{array}{c} .05784 \begin{array}{c} 1_{16} \\ .07100 \\ .08416 \\ .08416 \\ .09732 \\ .0$	$\begin{array}{c} .110 \\ .136 \\ \frac{7}{16}\frac{4}{3} + \\ .162 \\ \frac{7}{3}\frac{2}{3} + \\ .188 \\ \frac{7}{16}\frac{7}{3} - \\ .240 \\ \frac{7}{16}\frac{7}{3}\frac{1}{3} + \\ .266 \\ \frac{7}{16}\frac{7}{3}\frac{1}{3} + \\ .292 \\ \frac{7}{3}\frac{7}{3}\frac{1}{3}\frac{1}{3} + \\ .344 \\ \frac{1}{3}\frac{7}{3}\frac{7}{3}\frac{7}{3} + \\ .376 \\ \frac{7}{3}\frac{7}{3}\frac{1}{3} + \\ .376 \\ \frac{7}{3}\frac{7}{3}\frac{7}{3} + \\ .376 \\ \frac{7}{3}\frac{7}{3} $	32 28 26 24 22 20 18 16 15 14 13 12 11 11 10	16 17 18 19 20 21 22 23 24 25 26 27 28 29 30	.26840	-526 -5528 -5578 -604 -630 -658 -734 -658 -734 -658 -734 -746 -748 -748 -748 -748 -748 -748 -748 -748	9 9 8 8 8 8 7 7 7 6 6 6 6 6

U. S. NAVY BOILER RIVETS

1		A	В	С	D	Weight of 10 heads	Weight per inch of shank L
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	Ç Ç L	$\begin{array}{c} \frac{9}{16} \\ \frac{5}{5} \\ \frac{11}{16} \\ \frac{3}{4} \\ \frac{13}{16} \\ \frac{7}{8} \\ \frac{15}{15} \\ \frac{1}{16} \\ \frac{1}$	$\begin{array}{cccccccccccccccccccccccccccccccccccc$	16 5 8 5 5 11 1 1 1 1 1 1 1 1 1 1 1 1 1 1	$\begin{array}{c} \mathbf{e} \\ $.531 .713 I.007 I.372 I.551 2.033 -2.258 2.871 3.584 3.910 4.761 5.170 6.215 7.391 8.490 9.941 II.507	.0556 .0704 .0869 .1052 .1251 .1470 .1703 .1956 .2225 .2512 .2816 .3137 .3477 .3833 .4207 .4599 .5006
	<u></u> <u>*</u>	I_{16}^{11}	3	I 3/8	I 11/16	17.300	.6336

U. S. NAVY HULL AND TANK RIVET HEADS



	Pan-h	.ead		Button				Count	ersunk	
A	В	С	D	A	В	С	A	В	С	D
1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	$\begin{array}{c} \frac{7}{16} \\ \frac{5}{8} \\ \frac{13}{16} \\ I \\ I \\ \frac{7}{16} \\ I \\ \frac{7}{16} \\ I \\ \frac{15}{16} \\ $	36 5 16 38 7 16 12 9 16 5 8 11 1 8	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	14 388 122 588 324 1788 I I I I I I I I I I I I I I I I I I	1 3 16 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	$\begin{array}{c} \frac{3}{16} \\ \frac{5}{16} \\ \frac{5}{16} \\ \frac{3}{8} \\ \frac{7}{16} \\ \frac{1}{2} \\ \frac{9}{16} \\ \frac{5}{8} \\ \frac{11}{16} \\ \frac{3}{4} \end{array}$	14 350 12 52 314 730 I 15 14	35 58 13 16 116 155 176 158 134 129 132	32 737 14 28 12 116 150 16	60 60 60 60 45 45 37 37

LENGTH OF ROUND HEAD RIVETS FOR DIFFERENT THICKNESSES OF METAL

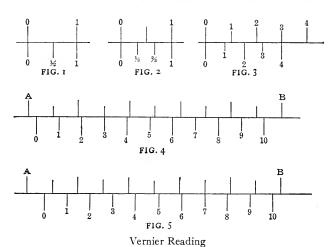
To find the required length of a rivet when thickness of metal between rivet heads is given, assuming the rivet hole to be $\frac{1}{15}$ inch larger than the rivet before it is heated, refer to the table below. Grip in inches means thickness of metal between rivet heads.

		Di	ameter in Inch	es	
Grip in Inches.	1/2	<u>5</u>	3	7 8	I
	`	L	ength in Inches	s.	
1 1 1 1 1 1 1 1 1 1 2 2 2 2 2 2 2 2 3 3 3 3		다마다의 나타나에 하나에 다시하는 나타나는 나타나는 나타나는 나타나는 나타나는 나타나는 나타나는 나타	TIS	- তিন্ধতিটন্থতিট্নত্ত্ৰিল্ল-প্ৰতিন্ত্ৰত্ত্ত্ত্ত্ত্ত্ত্ত্ত্ত্ত্ত্ত্ত্ত্ত্ত	165-14606-12006694768 1445616-12006694768 165-14606-12006768 146-14606-12006694768 165-14606-120066768 146-14606-12006694768 165-14606-14606-12006694768 165-14606-12006694768 165-14606-12006694768 165-14606-12006694768 165-14606-12006694768 165-14606-12006694768 165-14606-12006694768 165-14606-12006694768 165-14606-12006694768 165-14606-12006694768 165-14606-12006694768 165-14606-12006694768 165-14606-12006694768 165-14606-12006694768 165-14606-12006694768 165-14606-12006694768 165-14606-12006694768 165-14606-12006694768 165-14606-1200669476 165-14606-146069476 165-14606-146069476 165-14606-146069476 165-14606-146069476 165-14606-146069476 165-146069676 165-146069676 165-146069676 165-146069676

CALIPERING AND FITTING

THE VERNIER AND HOW TO READ IT

This method of measuring or of dividing known distances into very small parts is credited to the invention of Pierre Vernier in 1631. The principle is shown in Figs. 1 to 3 and its application in Figs. 4 and 5. In Figs. 1 and 2 both distances o-1 are the same but they are divided into different divisions. Calling o - 1 = 1 inch then in Fig. 1 it is clear that moving the lower seal one division will divide



the upper one in half. In Fig. 2 the upper scale is divided in half and the lower one in thirds. If the lower scale is moved either way until $\frac{1}{3}$ or $\frac{2}{3}$ comes under the end line, it has moved $\frac{1}{3}$ of an inch but if either of these are moved to the center line then it is only moved $\frac{1}{2}$ of this amount or $\frac{1}{8}$

Figure 3 shows the usual application of the principle except that it is divided in four parts instead of ten. Here both the scales have four parts but on the lower scale the four parts just equal three parts of the upper scale. It is evident that if we move the lower scale so that 0 goes to 1 and 4 goes to 4 that it will be moved $\frac{1}{4}$ the length of the distance o-4 on the upper scale. If this distance was 1 inch, each division on the upper scale equals $\frac{1}{4}$ inch and moving the lower scale so that the line 1 just matches the line next to 0 on the upper scale gives $\frac{1}{4}$ of one of these divisions or $\frac{1}{16}$ of an inch.

Figures 4 and 5 show the usual application in which the lower or vernier scale is divided into 10 parts which equals 9 parts of the upper scale. The same division holds good, however, and when the lower scale is moved so that the first division of the vernier just matches the first line of the scale, it has been moved just one tenth of a division. In Fig. 4 the third lines match so that it has moved $\frac{1}{10}$ and in Fig. 5, $\frac{1}{10}$ of a division. So if A B is one inch then each division is $\frac{1}{10}$ of an inch and each line of the vernier is $\frac{1}{10}$ of that or $\frac{1}{10}$ of an inch.

To find the reading of any vernier, divide one division of the upper or large scale by the number of divisions in the small scale. So if we had a vernier with 16 divisions in each, the large scale being 1 inch long, then the movement of one division is $\frac{1}{15}$ of $\frac{1}{15}$ or $\frac{1}{15}$ of

an inch.

READING THE MICROMETER

The commercial micrometer consists of a frame, the anvil or fixed measuring point, the spindle which has a thread cut 40 to the inch on the portion inside the sleeve or barrel and the thimble which goes outside the sleeve and turns the spindle. One turn of the

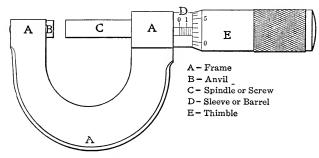


Fig. 6. — Micrometer

screw moves the spindle $\frac{1}{40}$ or .025 of an inch and the marks on the sleeve show the number of turns the screw is moved. Every fourth graduation is marked 1, 2, 3, etc., representing tenths of an inch or as each mark is .025 the first four means .025 \times 4 = .100, the third means .025 \times 4 \times 3 = .300.

means .025 \times 4 \times 3 = .300. The thimble has a beveled edge divided into 25 parts and numbered 0, 5, 10, 15, 20 and to 0 again. Each of these mean $\frac{1}{25}$ of a turn or $\frac{1}{25}$ of $\frac{1}{40} = \frac{1}{1000}$ of an inch. To read, multiply the marks on the barrel by 25 and add the graduations on the edge of the thimble. In the cut there are 7 marks on the sleeve and 3 on the thimble so we say $7 \times 25 = 175$, plus 3 = 178 or .178.

In shop practice it is common to read them without any multiplying by using mental addition. Beginning at the largest number shown on the sleeve and calling it hundreds and add 25 for each mark, we say in the case show 100 and 25, 50, 75 and then add the numbers shown on the thimble 3, making .178 in all. If it showed 4 and one mark, with the thimble showing 8 marks, the reading would be 400 + 25 + 8 = 433 thousandths or .433.

THE TEN-THOUSANDTH MICROMETER

This adds a vernier to the micrometer sleeve or barrel as shown in Fig. 7, which is read the same as any vernier as has been explained. First note the thousandths as in the ordinary micrometer and then look at the line on the sleeve which just matches a line on

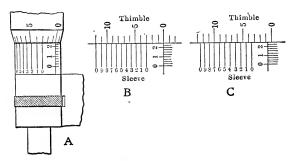


Fig. 7. - Micrometer Graduations

the thimble. If the two zero lines match two lines on the thimble, the measurement is in even thousandths as at B which reads .250. At C the seventh line matches a line on the thimble so the reading is .2507 inch.

MEASURING THREE-FLUTED TOOLS WITH THE MICROMETER

THE sketch, Fig. 8 on page 319, shows a V-block or gage for measuring three-fluted drills, counterbores, etc.

The angle being 60 degrees, the distances A, B, and C are equal. Consequently to determine the correct diameter of the piece to be measured, apply the gage as indicated in the sketch and deduct one third of the total measurement.

The use of this gage has a decided advantage over the old way of soldering on a piece of metal opposite a tooth or boring out a ring to fit to.

Using a standard 60-degree triangle for setting and a few different sizes of standard cylindrical plug gages for testing, the V-block may be easily and very accurately made.

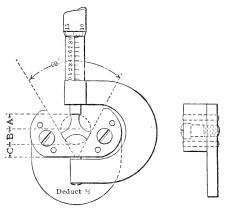


Fig. 8. — Measuring Three-Fluted Tools

PRESS AND RUNNING FITS

Parallel Press, Drive and Close Fits

Table 1, page 320, gives the practice of the C. W. Hunt Company, New York, for press, drive and close or hand fits for parallel shafts ranging between one and ten inches in diameter. In accordance with general practice, the holes for all parallel fits are made standard, except for unavoidable variation due to the wear of the reamer, the variation from standard diameter for the various kinds of fits being made in the shaft. This variation is, however, not positive, but is made between limits of accuracy or tolerance. Taking the case of a press fit on a two-inch shaft, for example, it will be seen that the hole—that is, the reamer—is kept between the correct size and 0.002 inch below size, while the shaft must be between 0.002 and 0.003 inch over size. For a drive or hand fit the limits for the hole are the same as for a press fit, while the shaft in the former case must be between 0.001 and 0.002 large and in the latter between 0.001 and 0.002 small.

Parallel Running Fits

Table 2, page 321, gives in the same way the allowances made by the same concern for parallel running fits of three grades of closeness. The variations allowed in the holes are not materially different from those of the preceding table, but the shafts are, of course, below instead of above the nominal size.

In all cases the tables apply to steel shafts and cast-iron wheels or other members. In the right-hand columns of the tables the formulas from which the allowances are calculated are given, and from

which the range of tables may be extended.

Table 1. Limits to Diameters of Parallel Shafts and Bushings (Shafts Changing)

						,	,
Diameters		ı in.	2 in.	3 in.	4 in.	5 in.	Formula:
Press Fit	$Shaft \Big \{$	++	++ .002	++ .003	+ .004	++ .005	(100. + b 100.) + (100. + b 100.) + (100. + b 100.)
Drive Fit	Shaft {	+ .0005	++.001	+ .0015	++	+ .0025	+ (.0005 d + .000) + (.0005 d + .001)
Hand Fit	$\text{Shaft} \Big \{$.00I	00I	00I	1.002	1.002	
All Fits	Hole {	+ 1.000	+ .000	+ .000	+ .000	1 + .000	
Diameters		6 in.	7 in.	8 in.	9 in.	ro in.	Formula:
Press Fit	$\mathrm{Shaft} \Big\}$	200° +	+ .007	800° + +	000. + 010. +	010 +	(000. + b 100.) + (100. + b 100.) +
Drive Fit	$Shaft \Big\}$	+ .003	+ .0035	++ .004	+ .0045	++ .005	+ (.0005 d + .000) + (.0005 d + .001)
Hand Fit	$\text{Shaft} \Big\}$	1.002	1.003	.003 .004	003 004	1.003	
All Fits	Hole $\left\{ ight.$	+ .000	+ .000 + .004	+ .000 004	+ .000 + .004	+ .000	

Table 2. Limits to Diameters of Parallel Journals and Bearings (Journals Changing)

			6			maar) aa.	
Diameters		ı in.	2 in.	3 in.	4 in.	5 in.	Formula:
Close Fit	$Shaft\Big\{$	1.003	1.006	005 007	900.	700. –	- (.oo. d + .oo2) - (.oo. d + .oo4)
Free Fit	$\mathrm{Shaft}\Big\{$	008 011	0009	010	.011	012	(oo. + b 100.) - (oo. + b 100.) -
Loose Fit	Shaft {	023 028	026 031	029 034	032	035	- (.003 d + .020) - (.003 d + .025)
All Fits	Hole {	+ .000	+ .000	+ .000	+ 1	+ .000	
Diameters		6 in.	7 in.	8 in.	9 in.	ro in.	Formula:
Close Fit	Shaft $\left\{ ight.$	1 .008 0 .010	000. –	010. —	011 013	.012	- (.oo. d + .oo2) - (.oo. d + .oo4)
Free Fit	Shaft {	or3 or6	o.4 o.7	- 015	610. –	.017	(o10, + b 100,) - (o10, + b 100,) -
Loose Fit	Shaft {	038 043	04I	044	047	1.050	- (.003 d + .020) - (.003 d + .025)
All Fits	Hole {	+ .000	+ .000	+ - 000. + 00.	+ I 000. 400.	+ 1 000. 1	

Shrink Fits

Table 3 gives the practice of the General Electric Company, Schenectady, New York, in regard to shrink fits, the same allowances also being made for press fits on heavy work such as couplings, etc.

Dia. In.	Allowance	Dia. In.	Allowance	Dia. In.	Allowance
I 2	.001 .0015	20 22	.008 .0088	42	.0143
3	.0015	24 26	.0093	44 46 48	.015 .0155 .016
6 8	.0035	28	.0105	60	.020
10 12	.0053	32 34	.0115 .012	72 84 96	.027 .030
14 16	.0065 .007	36 38	.0128 .0133	108	.033 .0355
18	.0075	40	.0138	132 144	.038 .040

TABLE 3. ALLOWANCES FOR SHRINK FITS

LIMITS FOR GAGES

The Newall Engineering Company, when developing their system of limit gages, investigated the practice of the leading English, Continental and American engineering concerns relative to allowances for different kinds of fits and prepared a table which is the average of all the data received, every point included being covered by the practice of some prominent establishment. The limits and allowances thus arrived at for shop gages are given in Table 4, which is self-explanatory.

Table 4. Limits and Allowances in Shop Gages for Different Kinds of Fits

Nominal Diameters	1"	1"	2"	· 3"	4"	5″	6"
Over size	.00025	.00025	.00025	.00050	.00050	.00050	.00050

Limits in Plug Gages for Standard Holes

(Table Continued on Page 323)

TABLE 4 Continued. — LIMITS IN SHOP GAGES
Allowances — over Standard — for Force Fits

Allowances — over Standard — for Force Fits								
Nomi	nal Diameter	5 1 1	1"	2"	3"	4"	5"	6"
Mean. High Low Margin		0010	5.00175 0.00200 0.00150 0.00050	.00100	.00600	.00800	.01000	.01200
	Allowar	nces — o	ver Stan	dard –	-for D	riving	Fits	
Nomina	al Diameters	1"	1"	2"	3"	4"	5"	6"
High Low Margin		.000375 .00050 .00025 .00025	.00100	.00150	.00250 .00150 .00100	.00250 .00300 .00200	.00350 .00250 .00100	.00100
Allowances — Below Standard — for Push or Keying Fits								
Nomina	al Diameters	1" §"	ı"	2"	3"	4"	5"	6"
High Low Margin		.00025 .00075 .00050	.00050	.00150	.00200	.00250	.00250	.00250
		Clear	ances for	Runn	ing Fits	S		
Class of Gage	Diameters	12"	I	2"	3"	4"	5"	6"
х.	Mean High Low Margin	.00150 .00100 .00200	.00200 .00125 .00275	.00175	.00200	.00380 .00250 .00500	.00300	.00350
Y .	Mean High Low Margin	.00100 .00075 .00125	.00150	.00190 .00125 .00250	.00230 .00150 .00300	.00270 .00200 .00350	.00310 .00225 .00400	.00350 .00250 .00450
Z.	Mean High Low Margin	.000625 .00050 .00075	.00100	.001 20 .000 7 5 .001 50	.00140 .00100 .00200	.00160 .00100 .00225	.00180 .00125 .00250	.00200 .00125 .00275
	Chargin	.00025	.00050	.00075	.00100	.00125	.00125	.00150

Class X is suitable for engine and other work requiring easy fits. Class Y is suitable for high speeds and good average machine work. Class Z is suitable for fine tool work.

Limits For Work Ground to Various Classes of Fits

Table 5 gives the limits used by the Brown and Sharpe Manufacturing Company in grinding work to various classes of fits required in machine manufacture.

TABLE 5 — GRINDING LIMITS FOR CYLINDRICAL PIECES AS ADOPTED BY BROWN AND SHARPE MFG. CO.

RUNNING FITS -- ORDINARY SPEED

Small

Small

Small

Small Small

To ½-inch diameter, inc. 0.00025 to 0.00075

To 1-inch diameter, inc. 0.00075 to 0.0015

To 2-inch diameter, inc. o.oo15 to o.oo25

To \frac{1}{2}-inch diameter, inc.	RUNNING FITS — HIGH SPEED, HEAVY PI	RESSURE	AND ROCKER	SHAFTS
To \frac{1}{2}-inch diameter, inc.	To 1-inch diameter, inc. To 2-inch diameter, inc. To 3½-inch diameter, inc.	0.001 0.002 0.003	to 0.002 to 0.003 to 0.0045	Small Small Small
To 1-inch diameter, inc.	SLIDING FIT	rs.		
To \frac{1}{2}-inch diameter, inc.	To 1-inch diameter, inc. To 2-inch diameter, inc. To 3½-inch diameter, inc.	0.0005 0.001 0.002	to 0.001 to 0.002 to 0.0035	Small Small Small
To 1-inch diameter, inc. Standard to 0.0005 Small To 2-inch diameter, inc. Standard to 0.001 Small To 3½-inch diameter, inc. Standard to 0.001 Small To 3½-inch diameter, inc. Standard to 0.002 Small To 6-inch diameter, inc. Standard to 0.002 Small DRIVING FITS — FOR SUCH PIECES AS ARE REQUIRED TO BE READILY TAKEN APART To ½-inch diameter, inc. Standard to 0.00025 Large To 1-inch diameter, inc. 0.00025 to 0.0005 Large To 2-inch diameter, inc. 0.0005 to 0.0007 Large To 3½-inch diameter, inc. 0.0007 to 0.001 Large To 6-inch diameter, inc. 0.001 to 0.001 Large To 1-inch diameter, inc. 0.0005 to 0.001 Large To 1-inch diameter, inc. 0.0005 to 0.001 Large To 1-inch diameter, inc. 0.0001 to 0.002 Large To 2-inch diameter, inc. 0.001 to 0.002 Large To 2-inch diameter, inc. 0.002 to 0.003 Large To 3½-inch diameter, inc. 0.0002 to 0.003 Large To 3½-inch diameter, inc. 0.0002 to 0.003 Large To 3½-inch diameter, inc. 0.0003 to 0.004 Large	STANDARD FI	ITS		
TAKEN APART To ½-inch diameter, inc. Standard to 0.00025 Large To 1-inch diameter, inc. 0.00025 to 0.0005 Large To 2-inch diameter, inc. 0.0005 to 0.00075 Large To 3½-inch diameter, inc. 0.00075 to 0.001 Large DRIVING FITS To ½-inch diameter, inc. 0.0005 to 0.001 Large To 1-inch diameter, inc. 0.0005 to 0.001 Large To 2-inch diameter, inc. 0.0001 to 0.002 Large To 2-inch diameter, inc. 0.0002 to 0.002 Large To 3½-inch diameter, inc. 0.0002 to 0.003 Large To 3½-inch diameter, inc. 0.0003 to 0.004 Large	To 1-inch diameter, inc. To 2-inch diameter, inc. To 3½-inch diameter, inc.	Standar Standar Standar	rd to 0.0005 rd to 0.001 rd to 0.0015	Small Small Small
To 1-inch diameter, inc.		-	IRED TO BE R	EADILY
To $\frac{1}{2}$ -inch diameter, inc. 0.0005 to 0.001 Large To 1-inch diameter, inc. 0.001 to 0.002 Large To 2-inch diameter, inc. 0.002 to 0.003 Large To $3\frac{1}{2}$ -inch diameter, inc. 0.003 to 0.004 Large	To 1-inch diameter, inc. To 2-inch diameter, inc. To 3½-inch diameter, inc.	0.00025 0.0005 0.00075	to 0.0005 to 0.00075 to 0.001	Large Large Large
To r-inch diameter, inc. 0.001 to 0.002 Large To 2-inch diameter, inc. 0.002 to 0.003 Large To $3\frac{1}{2}$ -inch diameter, inc. 0.003 to 0.004 Large	DRIVING FIT	rs		
	To 1-inch diameter, inc. To 2-inch diameter, inc. To 3½-inch diameter, inc.	0.001 0.002 0.003	to 0.002 to 0.003	Large Large

Table 5 Continued

FORCING FITS

To ½-inch diameter, inc. To 1-inch diameter, inc. To 2-inch diameter, inc. To 3½-inch diameter, inc.	0.0015	to 0.0025	Large
	0.0025	to 0.004	Large
	0.004	to 0.006	Large
To 6-inch diameter, inc	0.006	to 0.009	Large

SHRINKING FITS — FOR PIECES TO TAKE HARDENED SHELLS $\frac{3}{8}$ INCH THICK AND LESS

SHRINKING FITS — FOR PIECES TO TAKE SHELLS, ETC., HAVING A THICKNESS OF MORE THAN $\frac{3}{8}$ INCH

To $\frac{1}{2}$ -inch diameter, inc	0.0005	to 0.001	Large
To 1-inch diameter, inc	100.0	to 0.0025	Large
To 2-inch diameter, inc.	0.0025	to 0.0035	Large
To $3\frac{1}{2}$ -inch diameter, inc	0.0035	to 0.005	Large
To 6-inch diameter, inc	0.005	to 0.007	Large

GRINDING LIMITS FOR HOLES

To ½-inch diameter, inc. To 1-inch diameter, inc. To 2-inch diameter, inc. To 3½-inch diameter, inc.	Standard to 0.00075 Standard to 0.001 Standard to 0.0015	Large Large Large Large
To 6-inch diameter, inc. To 12-inch diameter, inc.		Large Large

Metric Allowances For Fits of All Classes

Table 6 covers allowances worked out by the Newall Engineering Company for use in connection with metric measurements; the allowances being given in decimals of a millimeter.

The Newall system is based on a hole "basis," which means that all holes are produced as near the standard size as commercially possible and the allowances are made in the shaft or other fitting. The first part of the table shows the tolerances allowable in a standard hole for two grades of work which are designated by classes A and B. The remainder of the table covers fits of various classes.

TABLE 6 - THE NEWALL STANDARD Tables of Allowances for Various Classes of Fits in Millimeters

	inces in			for Two			imeters
Nominal Dia.	Up to 15	16-25	26-50	51-75	76-100	101-125	126-150
High limit Low limit Tolerance	+ 0.007 - 0.007 0.014	+ 0.013 - 0.007 0.020	+ 0.019 - 0.007 0.026	+ 0.026 - 0.013 0.039	+ 0.026 - 0.013 0.039	+ 0.026 - 0.013 0.039	+ 0.03 - 0.01 0.05
High limit Low limit Tolerance	+ 0.013 - 0.013 0.026	+ 0.019 - 0.013 0.032	+ 0.026 - 0.013 0.039	+ 0.032 - 0.019 0.052	+ 0.039 - 0.019 0.058	+ 0.045 - 0.019 0.064	+0.05 - 0.02 0.07
		Force	and Sh	rink Fits	3	У	
Nominal Dia.	Up to 15	16-25	26-50	51-75	76-100	101-125	126-15
High limit Low limit Tolerance	+ 0.026 + 0.013 0.013	+ 0.051 + 0.038 0.013	+ 0.102 + 0.077 0.025	+ 0.153 + 0.115 0.038	+ 0.204 + 0.152 0.052	+ 0.203	+ 0.30 + 0.25 0.05
		1	Oriving 1	Fits			
Nominal Dia.	Up to 15	16-25	26-50	51-75	76-100	101-125	126-15
High limit Low limit Tolerance	+ 0.013 + 0.007 0.006	+ 0.026 + 0.019 0.007	+ 0.039 + 0.026 0.013	+ 0.064 + 0.039 0.025	+ 0.077 + 0.051 0.026	+ 0.089 + 0.063 0.02t	+ 0.10 + 0.07 0.02
Pt	ush Fits	— Go ir	Easy b	ut will r	not Turr	1	'
Nominal Dia.	Up to 15	16-25	26-50	51-75	76-100	101-125	126-15
High limit Low limit Tolerance	- 0.006 - 0.019 0.013	- 0.006 - 0.019 0.013	- 0.006 - 0.019 0.013	- 0.012 - 0.026 0.014	- 0.012 - 0.026 0.014	- 0.012 - 0.026 0.014	- 0.01 - 0.02 0.01
	Running	Fits fo	r Engine	and Sir	nilar W	ork	
Nominal Dia.	Up to 15	16-25	26-50	51-75	76-100	101-125	1,26-15
High limit Low limit Tolerance	- 0.025 - 0.051 0.026	- 0.032 - 0.070 0.038	- 0.045 - 0.090 0.045	- 0.051 - 0.108 0.057	- 0.063 - 0.127 0.064	- 0.076 - 0.146 0.070	- 0.08 - 0.16
· R	unning I	its for (Good Av	erage M	achine	Work	
Nominal Dia.	Up to.15	16-25	26-50	51-75	76-100	101-125	126-15
High limit Low limit Tolerance	- 0.019 - 0.032 0.013	- 0.025 - 0.051 0.026	- 0.032 - 0.064 0.032	- 0.038 - 0.076 0.038	- 0.051 - 0.089 0.038	- 0.057 - 0.101 0.044	- 0.06 - 0.11 0.05
	Rui	nning Fi	ts for Fi	ine Tool	Work		
Nominal Dia.	Up to 15	16-25	26-50	51-75	76-100	101-125	126-15
High limit	- 0.012	- 0.019			- 0.025 - 0.057	- 0.032 - 0.064	

- 0.039

0.020

Low limit

Tolerance....

- 0.019

0.007

- 0.032

0.013

- 0.051

0.026

- 0.057

0.032

- 0.064

0.032

- 0.070

0.038

Press Fits For Wheel Hubs

The practice of the Boston Elevated Railroad is to allow 8 tons per inch of diameter. An excess of 2 tons total pressure is allowed for cast iron, while the minimum pressure may be from 8 to 13 tons below the normal according to diameter, as shown by the following table. These are for cast iron hubs with the cone 5 inches in diameter $-7\frac{1}{2}$ inches long. For cast steel or wrought iron 20 per cent greater pressure is allowed.

Diameter of Fit	Minimum Pressure	Maximum Pressure	Variation Allowed in Tons
$\begin{array}{cccccccccccccccccccccccccccccccccccc$	12 16 20 24 28	22 26 30 34 38	10
5 to $5\frac{7}{16}$	31	42	11
5\frac{1}{2} to $5\frac{15}{16}$	35	46	
6 to $6\frac{7}{16}$	38	50	I 2
$6\frac{1}{2}$ to $6\frac{1}{16}$	42	54	
7 to $7\frac{7}{16}$ to $7\frac{7}{16}$	45 49	58 62	13
8 to $8\frac{7}{16}$	52	66	1.1
$8\frac{1}{2}$ to $8\frac{15}{16}$	56	70	
9 to $9\frac{7}{16}$	59	74	15
$9\frac{1}{2}$ to $9\frac{1}{1}\frac{5}{1}$	63	78	
10 to $10\frac{7}{16}$	67	82	

RUNNING FITS FOR POWER TRANSMISSION MACHINERY

The Dodge Manufacturing Company has different standards for different classes of work for running fits. Their ordinary bearings vary from $\frac{1}{3^4}$ inch for 1 inch to a little over $\frac{1}{3^2}$ for 6 inches. Their clutch sleeves, which are babbitted, run very much closer, varying from 0.008 to about 0.0015 inch. Loose pulleys are sometimes made as close as 0.003 inch on the smaller sizes. The company has found that a good standard on loose pulleys is about $\frac{1}{64}$ on a $2\frac{1}{2}$ inch hole and varying proportionately for diameters above and below that. This is very much freer than most people recommend, but it has been found that in the general trade there is more difficulty in having a little too tight a fit than there is in having the fit a little too loose.

MAKING ALLOWANCES WITH THE CALIPER'S FOR RUNNING, SHRINK, AND PRESS FITS

One of the familiar devices of the machinist consists in giving the inside calipers a certain amount of side play, when it is desirable to obtain a measure minutely less than the full diameter of the hole, as in making a loose or running fit, or a sliding fit as of a plunger in a cylinder. Thus in Fig, 9, A is the diameter of the bore, B the caliper setting and C the side play permitted the caliper in the hole.

In the table below is given a list of the reduced dimensions for different amounts of side play of the calipers in a 12-inch hole. From this, the dimensions may be obtained for holes of other diameters by division. Where in the table the side play is 2 inches, if we divide the items by 4 we have the side play and the reduced dimension for a 3-inch hole, or 0.5 inch and 2.9894 inches respectively.

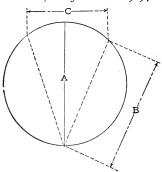


Fig. q. - Side Play of the Calipers

TABLE OF REDUCED DIAMETERS INDICATED BY INSIDE CALIPERS
FOR DIFFERENT AMOUNTS OF SIDE PLAY IN A 12-INCH HOLE
Side play

de	play																										
	0.1.															 					I	I	Ģ.	99	19	9	
	0.2.															 					Ι	I	٠,	99	19	Ι	
	0.4.															 					I	Ι	٠,	99)8,	3	
	o. 6.															 					Ι	1	Ç.	99	,6	2	
	0.8.													 		 		•			1	Ι	Q.	99	13.	3	
	1.0.													 		 				•	1	I	Q.	98	9.	5	
	1.2.													 		 				•	1	1	٠,	98	4	9	
	1.4.													 		 					I	1	Ģ.	7	9.	5	
	1.6.													 		 				•	Ι	Ι	Q.	7	3	0	
	1.8.													 	 					•	I	1	Ģ.)6	6	0	
	2.0.													 	 	 				•	Ι	I	Ģ.	95	7	9	
	2.2.													 	 	 				÷	1	Ι	Ç.)4	.90	0	
	2.4.												 	 	 	 				•	I	I	ç.)3	9	I	
	2.5.												 	 	 			•	٠	•	I	I	ç.)3	39	9	
	3.0.												 	 ٠,	 				•		Į	I	ç.	C	4	4	

Axial Inclination of the Calipers in Measuring for Shrink or Press Fits

In the case worked out on page 328, it was desired to produce a hole slightly larger than the piece to go into it, or a piece slightly smaller than the hole. In operations where a hole is wanted somewhat smaller than the piece to be shrunk or pressed into it, a similar plan of measuring can be employed, and a table giving the tightness can be computed. The sketch, Fig. 10, will serve to make the meaning clear. The distance A is the diameter of a hole and line a is the length of a gage the exact size of the piece to be pressed or shrunk into the hole. The distance b is the amount the gage lacks of assuming a position square or at a right angle to the axis of the hole.

It is an easy matter to make a table as suggested. It is only necessary to find the different lengths for the hypotenuse a for the right-angle triangle of which A is the constant base and b the perpendicular, taking b at different lengths from $\frac{1}{3}$ inch to 2 inches. Assuming the diameter to be 12 inches, then the lengths indicated for different inclinations in the direction of the axis will be as given

in the following table.

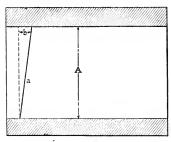


Fig. 10. - Inclination of the Calipers for Press Fits

TABLE FOR AXIAL INCLINATION OF CALIPERS IN ALLOWING FOR SHRINK OR FORCE FITS IN A 12-INCH HOLE

 Inclination of calipers
 12.00065

 \$\frac{1}{2} \text{ inch}\$
 12.00260

 \$\frac{3}{2} \text{ inch}\$
 12.00580

 \$\frac{1}{2} \text{ inch}\$
 12.01040

 \$\frac{3}{2} \text{ inch}\$
 12.01266

 \$\frac{3}{2} \text{ inch}\$
 12.03180

 \$\frac{1}{2} \text{ inch}\$
 12.03180

 \$\frac{1}{2} \text{ inch}\$
 12.04159

 \$\frac{1}{2} \text{ inch}\$
 12.03338

 \$\frac{1}{2} \text{ inch}\$
 12.03338

 \$\frac{1}{2} \text{ inches}\$
 12.12689

 \$\frac{1}{2} \text{ inches}\$
 12.10550

Side Play of Calipers in Boring Holes Larger than a Piece of Known Diameter

The following is an approximate rule for obtaining the variation in the size of a hole corresponding to a given amount of side play in the calipers. The rule has the merit of extreme simplicity and can be applied equally well to all diameters except the very smallest. In most cases the calculation is so simple that it can be done mentally without having recourse to pencil or paper.

The Calculation

Let A in Fig. 11 = side play of calipers or end measuring rod in sixteenths of an inch.

B = dimensions to which calipers are set, or length of measuring rod in inches.

C =difference between diameter of hole and length of B in thousandths of an inch.

Then $C = \frac{A^2}{2B}$, within a very small limit.

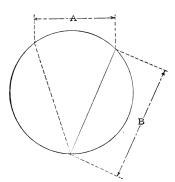


Fig. 11. — Caliper Side Play

Example: A standard end measuring rod, $5\frac{1}{2}$ inches long, has $\frac{3}{8}$ inch of side play in a hole. What is the size of the hole? In this case A=6 and $B=5\frac{1}{2}$. Apply the above formula:

 $C = \frac{6 \times 6}{11} = \frac{36}{11} = 3.27$ thousandths of an inch, or 0.00327 inch.

The diameter of the hole, therefore, is $5\frac{1}{2} + 0.00327$ or 5.50327. The method will be found to be correct within a limit of about 0.0002 inch if the amount of side play is not more than one eighth of the diameter of the hole for holes up to 6 inches diameter; within 0.0005 inch for holes from 6 inches up to 12 inches; and within 0.001 for holes from 12 inches up to 24 inches.

Allowing for Running and Driving Fits

This rule has been found to be useful for boring holes of large diameters in which allowances have to be made for running or driving fits, as only a single measuring rod for each nominal size is required. The rods should be of standard length, or a known amount less than standard, the allowances being obtained by varying the amount of side play when boring. The rule is also capable of determining limits, as the maximum and minimum amount of side play allowable can be specified. The measuring rods should be tapered at each end and the points slightly rounded. For accurate work, the body of the rod should be encased in some non-conducting material to nullify the effect of the heat of the hand.

In comparing this method with that described on page 330, it should be remembered that the conditions are reversed - that is to say, the first method is for setting calipers to a given dimension smaller than a hole of known diameter, whereas the method now described is for boring a hole a given amount larger than a gage of

known length.

In measuring the side play it is sufficient to take it to the nearest sixteenth of an inch, and if anything like accuracy is required it should be measured not guessed at.

DIMENSIONS OF KEYS AND KEY-SEATS

THE following rules and table on page 332, as prepared by Baker Bros., Toledo, Ohio, give dimension of keys and key-seats.

The width of the key should equal one fourth the diameter of the

The thickness of the key should equal one sixth the diameter of the shaft. The depth in the hub for a straight key-seat should be one half

the thickness of the kev.

The depth in the hub at the large end, for a taper key-seat, should

be three fifths the thickness of the key. The taper for all key-seats should be $\frac{3}{16}$ inch in I foot of length.

The depth to be cut in the hub for taper key-seats, at the large end, is greater than those cut straight, for the reason that unless this is done the depth in the hub at the small end will not be sufficient, especially in long key-seats.

The depths of key-seats in the table are given in thousandths of an inch and measured from the edge of the key-seat, and not from the center. In this manner the exact depth of key-seat can be

measured at any time after it is cut.

For extra long key-seats the depth cut in the hub may be slightly increased, but for the average work the table will be found correct.

DIMENSIONS OF KEYS AND KEY-SEATS. (BAKER BROS.)

Size of Hole								
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	Size of Hole	Equiv-	Width of	Size of	Thickness	Fractional	be Cut in Hub for Straight	Large End for Taper
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	т	Ι.	.25	1	.166	-3-	.003	.112
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$.265	į	1	16		
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	1 1 0		.281	į	.187	3		1
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	1 3 c			.5_		1,6 7		
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	т ‡			5.5		3.2 7.		
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	1.5			1 6 -5		3,2		
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	13 13			3		3 2 1	.125	
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	17			3		į	.125	
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	T 10			3		į	.125	
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	I 1 6	1.562		3	.26	1		
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	T 5			7		9		
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	111			77	.281	3 2		
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	T 3 0			77		9		
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	113			77		9 9		
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	17	1.875		10	.312	11		
$ \begin{array}{c ccccccccccccccccccccccccccccccccccc$	I 15			1/3	.323	11		
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$				1/2		11		
$ \begin{array}{c ccccccccccccccccccccccccccccccccccc$	$2\frac{1}{16}$	2.062		į		11		
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	21			1/2		11/2		
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	215	2.187		1/2		11		
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	21			į		11		
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	215		.578	1/2	.385	11		
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	$2\frac{3}{8}$			5	.396			.262
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	27			5	.406	77		.262
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	28		.625	5	.416	77		.262
$ \begin{array}{c ccccccccccccccccccccccccccccccccccc$	216	2.562	.641	5		75		.262
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	25	2.625		5		7.		.262
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	$2\frac{11}{16}$	2.687	.672	5 8		77		.262
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	$2\frac{\hat{3}}{4}$.687	5		7.6		.262
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	$2\frac{18}{16}$	2.812		5 8		7		.262
$\begin{array}{cccccccccccccccccccccccccccccccccccc$	$2\frac{7}{8}$	2.875	.719	3 4	.479	1/2	.25	•3
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	$2\frac{1}{1}\frac{5}{6}$	2.937		34	.49	$\frac{1}{2}$.25	•3
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	3	3.	.75	34	.5	$\frac{1}{2}$		•3
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	2 1	3.125	.781	34	.521	$\frac{1}{2}$		•3
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	3 1 6	3.187	.797	34	.531	$\frac{1}{2}$.25	•3
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	$3\frac{1}{4}$	3.25		34		$\frac{1}{2}$		•3
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	$3\frac{3}{8}$	3.375	.844	7/8	.562	<u>5</u>	.312	
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	$3\frac{7}{16}$	3.437	.859	$\frac{7}{8}$.573	<u>5</u>	.312	.375
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	$3\frac{1}{2}$	3.5	.875	$\frac{7}{8}$.583	<u>5</u> 8	.312	
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	38	3.625	.906	78		<u>5</u>	.312	.375
$ \begin{array}{c ccccccccccccccccccccccccccccccccccc$	$3\frac{11}{6}$	3.687		7/8		<u>5</u> 8	.312	.375
$ \begin{array}{c ccccccccccccccccccccccccccccccccccc$	$3\frac{3}{4}$	3.75	.937	7/8		<u>5</u> 8	.312	
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	3 7 _	3.875		I	.646	11 16		.412
$4 \mid 4. \mid 1. \mid 1 \mid .666 \mid \frac{11}{16} \mid .343 \mid .412$	$3\frac{15}{16}$	3.937	.984		.656	$\frac{11}{16}$		
	4	4.	I.	I	.666	$\frac{1}{1}\frac{1}{6}$	·343	.412

DIMENSIONS OF STRAIGHT KEYS

Another system of keys used by a good many manufacturers is given in the table following, the sizes of shafts ranging by sixteenths from $\frac{5}{16}$ inch to 4 inches and by eighths from 4 to 6 inches. The keys are square until the $1\frac{1}{3}$ inch shaft is reached, when the thickness of the key becomes $\frac{1}{16}$ less than the width. With the $4\frac{1}{4}$ size the thickness of the key becomes $\frac{1}{8}$ inch less than the width and this difference is constant up to the $5\frac{1}{3}$ shaft when the width exceeds the thickness by $\frac{3}{16}$ inch, this difference in the two dimensions continuing throughout the remainder of the table.

DIMENSIONS OF STRAIGHT KEYS

Dia. of Shaft	Width of Key	Thickness of Key	Dia. of Shaft	Width of Key	Thickness of Key	Dia. of Shaft	Width of Key	Thickness of Key	Dia. of Shaft	Width of Key	Thickness of Key
$\begin{array}{cccccccccccccccccccccccccccccccccccc$	3.23.35.180.180.180.103.103.103.103.103.1014.144.144.144.144.144.144.144.144.14	© 0 0 10 10 10 0 0 0 0 0 0 0 0 0 0 0 0 0	$\begin{array}{c} 1\frac{1}{4} \cdot \frac{1}{5} \cdot \frac{1}{6} \cdot \frac{1}{6} \cdot \frac{1}{1} \cdot \frac{1}{2} \cdot \frac{1}{6} \cdot \frac{1}{6} \cdot \frac{1}{1} \cdot \frac{1}{2} \cdot \frac{1}{1} $	5.65.65.75.765.765.765.765.767.767.767.7	$\frac{5.6}{1.5}$, $\frac{6.7}{1.5}$. The contraction of	2 2 2 2 2 2 2 2 2 3 3 3 3 3 3 3 3 3 3 3	+210,100,100,100,100,100,100,100,100,100,	7 10 102 102 103 103 103 103 103 103 103 103 103 103	3411178111 1811480 181478 1811480 18150 18	24 4 24 4 26 25 25 25 25 25 25 25 25 25 25 25 25 25	1-10-10-20-4-20-4-20-4-20-4-20-4-20-4-20

SQUARE FEATHER KEYS AND STRAIGHT KEY SIZES

The tables on page 334 give the sizes of square feather keys and regular straight keys in accordance with the practice of Jones & Laughlin, Pittsburg. For taper keys, this concern and many others use a $\frac{1}{3}$ -inch per foot taper.

SQUARE FEATHER KEY SIZES. (JONES & LAUGHLIN)

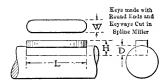
Dia. of Shaft	Size of Key	Dia. of Shaft	Size of Key
$\begin{array}{cccccccccccccccccccccccccccccccccccc$	1	316 to 38 316 to 38 316 to 35 316 to 35 316 to 36 316 to 42 416 to 42 416 to 51 516 to 51 516 to 61	$\begin{array}{c} \frac{18}{17} & \times & \frac{18}{17} \\ \frac{7}{17} & \times & \frac{18}{17} \\ \frac{7}{15} & \times & \frac{18}{16} \\ 1 & \times & 1 \\ 1$

STRAIGHT KEY SIZES. (JONES & LAUGHLIN)

Dia. of Shaft	Size of Key	Dia. of Shaft	Size of Key
I to I la 1 1 1 1 1 1 1 1 1	** ** ** ** ** ** ** ** ** ** ** ** **	31/6 to 38/8 31/6 to 38/8 31/6 to 38/8 31/6 to 38/8 31/6 to 48/8 41/6 to 48/8 41/6 to 51/6 to 61/8 51/6 to 61/8	1



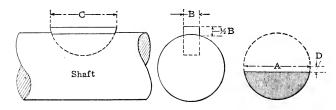
No. of Key	w	w	D
1 2 3 4 5	1855 93 16 14 5 16	.132 .165 .199 .264	15 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8



The Length "L" may vary from the table given, but must at least be equal to $(2 \times W)$. The maximum length of slot which can be cut in the Spline Milling machine in one cut; is (4'' + W). Note that the Width (W) is in all cases equal to the depth (D).

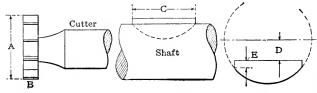
PRATT & WHITNEY KEY SYSTEM

Key No.	L	w	Н	D	Key No.	L	W	Н	D
1 2 3	$rac{1}{2}$		$ \begin{array}{r} \frac{3}{32} \\ \frac{9}{64} \\ \frac{3}{16} \end{array} $	$\begin{array}{c c} & 1 \\ \hline & 1 & 6 \\ \hline & 3 & 2 \\ \hline & 3 & 2 \\ \hline & 1 & 8 \end{array}$	22 23 F	I 3/8	$\begin{array}{c} \frac{1}{4} \\ \frac{5}{16} \\ \frac{3}{8} \end{array}$	$\begin{array}{c} \frac{3}{8} \\ \frac{1}{8} \\ \frac{1}{3} \\ \frac{5}{2} \\ \frac{9}{16} \end{array}$	$\frac{\frac{1}{4}}{\frac{5}{16}}$
4 5 6	<u>5</u> 8	$ \begin{array}{r} \frac{3}{32} \\ \frac{1}{8} \\ \frac{5}{32} \end{array} $	$\begin{array}{c} 9 \\ \overline{64} \\ \underline{3} \\ \overline{16} \\ \underline{15} \\ \overline{64} \end{array}$	$\frac{3}{3}\frac{1}{2}$ $\frac{1}{8}$ $\frac{5}{3}$	24 25 G	$1\frac{1}{2}$	14 5 16 38	$ \begin{array}{r} \frac{3}{8} \\ \frac{15}{32} \\ \frac{9}{16} \end{array} $	1 4 5 16 3 8
7 8 9	$\frac{3}{4}$	$ \begin{array}{r} $	$\begin{array}{c} \frac{3}{16} \\ \frac{1}{6} \\ \frac{1}{6} \\ \frac{9}{3} \\ 2 \end{array}$	$ \begin{array}{r} \frac{1}{8} \\ 5 \\ \hline 3 \\ \hline 1 \\ \hline 6 \end{array} $	51 52 53	$1\frac{3}{4}$	14 5 16 3 8	$\frac{\frac{3}{8}}{\frac{15}{32}}$	$\frac{\frac{1}{4}}{\frac{5}{16}}$
10 11 12 A	78	$\begin{array}{c} \frac{5}{32} \\ \frac{3}{16} \\ \frac{7}{32} \\ \frac{1}{4} \end{array}$	$\begin{array}{c} 155 \\ 64 \\ 9 \\ \hline 32 \\ 211 \\ 64 \\ 3 \\ 8 \end{array}$	$\begin{array}{c} \frac{5}{32} \\ \frac{3}{2} \\ 16 \\ \frac{7}{32} \\ \frac{1}{4} \end{array}$	26 27 28 29	2	$\begin{array}{c} 3 \\ \overline{16} \\ \underline{14} \\ 5 \\ \overline{16} \\ \underline{38} \end{array}$	$\begin{array}{c} 9 \\ \overline{3} \\ \underline{3} \\ 8 \\ 15 \\ \overline{3} \\ \underline{2} \\ \underline{4} \\ 9 \\ \overline{1} \\ \underline{6} \end{array}$	$ \begin{array}{c} 3 \\ 1 \\ 4 \\ 5 \\ 16 \\ 3 \\ 8 \end{array} $
13 14 15 B	I	$\begin{array}{c} \frac{3}{16} \\ \frac{7}{32} \\ \frac{1}{4} \\ \frac{5}{16} \end{array}$	$\begin{array}{c} 9 \\ \hline 3 \ 2 \\ 2 \ 4 \\ 6 \ 4 \\ 3 \ 8 \\ \underline{155} \\ 3 \ 2 \\ \end{array}$	$ \begin{array}{r} \frac{3}{16} \\ \frac{7}{32} \\ \frac{1}{4} \\ \frac{5}{16} \end{array} $	54 55 56 57	$2\frac{1}{4}$	$\frac{1}{4}$ $\frac{1}{5}$ $\frac{1}{16}$ $\frac{3}{8}$ $\frac{7}{16}$	3 8 5 5 2 9 6 1 2 8 2 8 2 8 2 8 2 8 2 8 2 8 2 8 2 8 2	$\frac{1}{4}$ $\frac{1}{5}$ $\frac{1}{16}$ $\frac{3}{8}$ $\frac{7}{16}$
16 17 18 C	I 1/8	$\begin{array}{c} \frac{3}{16} \\ \frac{7}{32} \\ \frac{1}{4} \\ \frac{5}{16} \end{array}$	$\begin{array}{c} 9 \\ \hline 3 \ 2 \\ 2 \ 4 \\ 6 \ 4 \\ \hline 3 \ 8 \\ 15 \ 2 \\ \hline 3 \ 2 \\ \end{array}$	$ \frac{3}{16} $ $ \frac{7}{32} $ $ \frac{1}{4} $ $ \frac{5}{16} $	58 59 60 61	$2\frac{1}{2}$	5 16 3 8 7 16 12	152 9 151 123 34	$\begin{array}{c} \frac{5}{16} \\ \frac{3}{8} \\ \frac{7}{16} \\ \frac{1}{2} \end{array}$
19 20 21 D E	I 1/4	$\begin{array}{c} \frac{3}{16} \\ \frac{7}{32} \\ \frac{1}{4} \\ \frac{5}{16} \\ \frac{3}{8} \end{array}$	9 82 82 83 85 85 85 85 86	$\begin{array}{c} \frac{3}{16} \\ \frac{7}{32} \\ \frac{1}{4} \\ \frac{5}{16} \\ \frac{3}{8} \end{array}$	30 31 32 33 34	3	38 7 12 9 16 58	9 12 3 3 4 7 2 3 5 6 1 2 3 1 1 2 3 1 1 1 1 1 1 1 1 1 1 1 1 1	$\begin{array}{c} \frac{3}{8} \\ \frac{7}{16} \\ \frac{1}{2} \\ \frac{9}{16} \\ \frac{5}{8} \end{array}$



WHITNEY KEYS AND CUTTERS. Nos. 1 TO 26 (Woodruff's Patent)

No. of Key and Cutter	Dia. of Cutter	Thick- ness of Key and Cutter	Length of Key	Key Cut Below Cen- ter	No. of Key and Cutter	Dia. of Cut- ter	Thick- ness of Key and Cutter	Length of Key	Key Cut Below Cen- ter
	A	В	С	D		A	В	С	D
1 2 3 4 5 6 7 8 9 10 11 12 A 13 14 15 B	1/21/22 - 1/22 5/23 5/23 5/23 5/23 5/24 5/4 5/4 5/4 5/4 5/25 7/25 7/25 7/25 7/25 7/25 7/25 7/25	1 1 0 0 0 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	1/21/21/CES/KES/KES/KES/KES/KES/KES/KES/KES/KES/K	64 8 8 16 16 16 16 16 16 16 16 16 16 16 16 16	16 17 18 C 19 20 21 D E 22 23 F	18-18-18-18-18-18-18-18-18-18-18-18-18-1	7 7 3 4 5 6 3 6 7 3 4 5 7 6 5 6 5 6 5 6 5 6 6 6 6 6 6 6 6 6 6	I I I I I I I I I I I I I I I I I I I	54 54 56 54 56 56 56 56 56 56 56 56 56 56 56 56 56



Note: Refer to table at top of page 337 for values of dimension E.

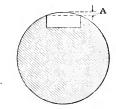
WHITNEY KEYS AND CUTTERS. Nos. 26 to 36

7 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	R-C-1-1000 C-1000 C-100	C 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	olsopisopisopisopisopisopisopisopisopisopi	$\alpha + \alpha +$	No. of Key and 2 2 2 3 3 3 4 3 5 3 6	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	* Thickness of Key and Cutter and Cutter	C 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	Center of the spin	Hat at End of E. S.

PROPORTIONS OF KEY HEADS

		(STAND	ARD GA	GE STEEL	co.)		
Ç	Å						
A	В	C	D	A	В	С	D
A 18 3 1-14-5 15 38 7 16-22 9 16 28 1-17 44 5 16 17 18 3 16 18 7 16 22 9 16 28 1-17 44 5 16 18 7 16 28 7 16 28 7 16 28 17 17 28 3 16 18 18 18 18 18 18 18 18 18 18 18 18 18	-100 cm 10 -144 5 10 cm 5 7 10 -45 5 10 cm 5 110 cm 4 cm 5 7 125 5 5 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	14************************************	7-20 NATION OF THE PROPERTY OF	1 1 1 1 1 1 1 2 2 2 2 2 2 2 2 2 2 2 2 2	I I I I I I I I I I I I I I I I I I I	2 2 3 3 3 3 3 3 3 4 4 4 4 4 4 4 5 5 5 5 5 5	$\begin{array}{cccccccccccccccccccccccccccccccccccc$

TABLE FOR FINDING TOTAL KEYWAY DEPTH

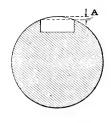


In the column marked "Size of Shaft" find the number representing the size; then to the right find the column representing the keyway to be cut and the decimal there is the distance A, which added to the depth of the keyway will give the total depth from the point where the cutter first begins to cut.

===	1	5 16	3	7.16	1/2
Size	1/4	16	S	16	2
Shaft	Keyway	Keyway	Keyway	Keyway	Keyway
$\begin{array}{cccccccccccccccccccccccccccccccccccc$	0.0325 0.0289 0.0254 0.0236 0.022 0.0198 0.0177 0.0164 0.0152 0.0131 0.0127 0.0123 0.012 0.0114 0.011 0.0107 0.0105 0.0099 0.0095 0.0093 0.0088 0.0083 0.0073	0.0413 0.0379 0.0346 0.0314 0.0283 0.0264 0.0228 0.021 0.0204 0.0198 0.0191 0.0185 0.0174 0.0164 0.0158 0.0147 0.0142 0.0136 0.0137 0.0147	0.0511 0.0465 0.042 0.0392 0.0365 0.0342 0.0319 0.0304 0.0279 0.0268 0.0254 0.0231 0.0221 0.0214 0.0207 0.0198 0.0199 0.0173 0.0168	0.0583 0.0544 0.0506 0.0476 0.0446 0.0421 0.0397 0.038 0.0364 0.0328 0.0328 0.0291 0.0282 0.0274 0.0265 0.0257 0.0253 0.0236 0.0229	0.067 0.0625 0.0581 0.0551 0.0522 0.0499 0.0477 0.0453 0.0429 0.0412 0.0395 0.0383 0.0371 0.0355 0.0328 0.0317 0.0308 0.0299
21/4	0.007	0.0107	0.0159	0.0216	0.0282

TABLE FOR FINDING TOTAL KEYWAY DEPTH

In the column marked "Size of Shaft" find the number representing the size; then to the right find the column representing the keyway to be cut and the decimal there is the distance A, which added to the depth of the keyway will give the total depth from the point where the cutter first begins to cut.



	1	Ī		1	
Size	14	15 15	38	176	1/2
Shaft	Keyway	Keyway	Keyway	Keyway	Keyway
2 2 2 2 2 2 2 2 2 2 2 2 3 3 3 3 3 3 3 3	0.0068 0.0066 0.0064 0.0063 0.0061 0.006 0.0059 0.0058 0.0057 0.0056 0.0054 0.0053 0.0052 0.0051 0.005 0.0049 0.0046 0.0045 0.0044 0.0044 0.0042 0.0041 0.0041	0.0104 0.0102 0.01 0.0098 0.0094 0.009 0.0089 0.0088 0.0084 0.0083 0.0081 0.008 0.0075 0.0075 0.0071 0.007 0.0066 0.0065 0.0065 0.0064 0.0062 0.0061	0.0155 0.0152 0.0149 0.0146 0.0142 0.0139 0.0136 0.0133 0.0129 0.0126 0.0112 0.0110 0.0114 0.0112 0.011 0.0108 0.0104 0.0102 0.0101 0.010 0.009 0.0098 0.0098 0.0097 0.0095 0.0094	0.0209 0.0202 0.0198 0.0194 0.0189 0.0185 0.0176 0.0172 0.0168 0.0164 0.0155 0.0155 0.0155 0.0149 0.0144 0.0138 0.0135 0.0135 0.0135 0.0132 0.0123	0.0274 0.0267 0.026 0.0253 0.0247 0.0242 0.0236 0.023 0.0226 0.0211 0.0207 0.0202 0.0198 0.0194 0.0191 0.0187 0.0188 0.0188 0.0174 0.0171 0.0168 0.0166 0.0163 0.0161 0.016

Tapers for Keys, etc., from $\frac{1}{16}$ to 1 inch per Foot. Amount of Taper for Lengths Varying by $\frac{1}{2}$ inch

r Foo					:	LENGTE	I				
Taper per Foot	ı	1 1/2	2	21/2	3	3 ¹ / ₂	4	41/2	5	51/2	6
1 6 18 3 6 1 14 5 6 3 8 7 6 12 9 6 5 8 1 6 3 4 3 6 7 8 5 6 1 1 1	.0104 .0156 .0208 .0260 .0312 .0364 .0416 .0468 .0520 .0572 .0625 .0677 .0729	.0156 .0234 .0312 .0390 .0468 .0546 .0702 .0780 .0858 .0938 .1016 .1094 .1172	.0208 .0312 .0416 .0520 .0625 .0729 .0833 .0937 .1041 .1145 .1250 .1354 .1458	.0260 .0390 .0520 .0650 .0780	.0312 .0468 .0625 .0781 .0937 .1093 .1250 .1406 .1562 .1718 .1875 .2031 .2187	.0364 .0546 .0729 .0911 .1092 .1275 .1457 .1641 .1823 .2004 .2188 .2370 .2552 .2734	.0416 .0625 .0833 .1041 .1250 .1458 .1666 .1875 .2083 .2291 .2500 .2708 .2916	.0468 .0703 .0937 .1171 .1406 .1640 .1874 .2109 .2343 .2577 .2813 .3047 .3281	.0520 .0781 .1041 .1302 .1562 .2083 .2343 .2604 .2864 .3125 .3385 .3645	.0572 .0859 .1145 .1432 .1718 .2004 .2291 .2577 .2864 .3150 .3438 .3724 .4010	.1250 .1562 .1875 .2187 .2500 .2812 .3125 .3437 .3750 .4062 .4375 .4687
r Foot]	LENGTH	τ				
Taper per Foot	61/2	7	71/2	8	81/2	9	91/2	10	101/2	11	1112
1 6 18 3 6 14 5 6 28 7 6 12 9 6 5 8 1 6 24 3 6 7 8 5 6 1 1 1 2 4 3 6 7 8 5 6 1 1 1 2 4 3 6 7 8 5 6 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	.0677 .1015 .1354 .1692 .2031 .2369 .2708 .3046 .3385 .3723 .4063 .4401 .4739 .5078	.0729 .1093 .1458 .1822 .2187 .2552 .2916 .3281 .3645 .4010 .4375 .4739 .5104	.0781 .1171 .1562 .2343 .2734 .3124 .3515 .3905 .4296 .4688 .5078 .5468	.0416 .0833 .1250 .1666 .2083 .2500 .2916 .3333 .3750 .4166 .4583 .5000 .5416 .5833 .6250 .6666	.0885 .1328 .1770 .2213 .2656 .3098 .3541 .3984 .4426 .4869 .5313 .5755 .6197	.0937 .1406 .1875 .2343 .2812 .3281 .3750 .4218 .4687 .5156 .5625 .6093 .6562	.0989 .1484 .1979 .2473 .2968 .3463 .3958 .4452 .4947 .5442 .5938 .6432 .6926	.1041 .1562 .2083 .2604 .3125 .3645 .4166 .4687 .5208 .5729 .6250 .6770 .7291 .7812	.1093 .1640 .2187 .2734 .3281 .3827 .4374 .4921 .5468 .6015 .6563 .7109 .7655 .8203	.1145 .1718 .2291 .2864 .3437 .4010 .4583 .5156 .5729 .6302 .6875 .7447 .8020 .8593	.1197 .1796 .2395 .2994 .3593 .4192 .4791 .5390 .5588 .7188 .7786 .8384 .8984

TAPERS AND DOVETAILS

MEASURING TAPERS

An Accurate Taper Gage

THE gage illustrated in Fig. 1 is an exceedingly accurate device for the gaging of tapers.

It is evident that if two round disks of unequal diameter are placed on a surface plate a certain distance apart, two straight-edges touching these two disks will represent a certain taper. It is also evident

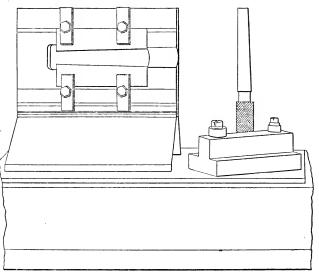


Fig. 1. — Accurate Taper Gage

that with the measuring instruments now in use it is a simple matter to measure accurately the diameters of the two disks, and the distance these disks are apart. These three dimensions accurately and positively determine the taper represented by the straight-edges touching the rolls. If a record is made of these three dimensions these conditions can be reproduced at any time, thus making it possible to duplicate a taper piece even though the part may not at the time be accessible.

The formulas on the following pages may be of service in connection with a general fall.

tion with a gage of this character:



Taper per Foot = 34"
FIG. 2

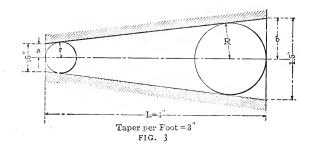
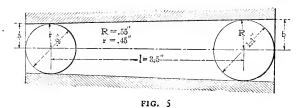




FIG. 4



Applications of Taper Gage

Formulas for Use in Connection with Taper Gage

To find Center Distance (1), refer to Fig. 2.

$$l = \frac{R - r}{t} \sqrt{1 + t^2}$$

To find Disk Diameters, refer to Fig. 3.

$$r = \frac{a}{L} \left[\sqrt{L^2 + (b - a)^2 + (b + a)} \right]$$

Dia. Small Disk = 2r

$$R = \frac{b}{L} \left[\sqrt{L^2 + (b-a)^2 - (b-a)} \right]$$

Dia. Large Disk = 2 R

To find Taper Per Foot (T), refer to Fig. 4.

$$T = 24 \left(\frac{R-r}{\sqrt{l^2 - (R-r)^2}} \right)$$

To find Width of Opening at Ends, refer to Fig. 5.

$$a = r\sqrt{\frac{l - (R - r)}{l + (R - r)}}$$

Width of opening at Small End = 2 a.

$$b = R\sqrt{\frac{l + (R - r)}{l - (R - r)}}$$

Width of Opening at Large End = 2 b.

Applications of Formulas

To Find Center Distance Between Disks

Suppose there are two disks as shown in Fig. 2, whose diameters are respectively $\mathfrak{1}^1_4$ and $\mathfrak{1}$ inch. It is desired to construct a taper of $\frac{3}{4}$ to the foot and the center distance l between disks must be determined in order that the gage jaws when touching both disks shall give that taper.

Let R = radius of large disk, or 0.625 inch.

r = radius of small disk, or 0.500 inch.

t =taper per inch on side, or

$$\frac{0.750}{24} = 0.03125$$
 inch.

Then

$$l = \frac{R - r}{t} \sqrt{1 + t^2} = \frac{0.125}{0.03125} \sqrt{1.000976} = 4 \times 1.0005 = 4.002 \text{ inches.}$$

To Find Disk Diameters

Suppose the gage jaws are to be set as in Fig. 3 for a three-inch per foot taper whose length is to be four inches. The small end is to be exactly $\frac{1}{2}$ inch and the large end for this taper will, therefore, be $\frac{1}{2}$ inches. What diameter must the disks be made so that when the jaws are in contact with them and the distance L over the disks measures 4 inches, the taper will be exactly three inches per foot? Here a represents $\frac{1}{2}$ the width of opening at the small end, and b one half the width of opening at the large end. The radius of the small disk may be found by the formula:

$$r = \frac{a}{L} \left\{ \sqrt{L^2 + (b-a)^2 + (b-a)} \right\}.$$

Then

$$r = \frac{0.250}{4} \left(\sqrt{16 + 0.25 + 0.5} \right)$$
$$= 0.0625 (4.0311 + 0.5) = 0.2832.$$

Diameter small disk = 0.2832 inch \times 2 = 0.5664 inch. For the large disk:

$$R = \frac{b}{L} \left\{ \sqrt{L^2 + (b-a)^2 - (b-a)} \right\}.$$

Then

$$R = \frac{0.75}{4} \left(\sqrt{16 + 0.25} - 0.5 \right)$$

= 0.1875 (4.0311 - 0.5) = 0.6621.

Diameter large disk = 0.6621 inch \times 2 = 1.3242 inches.

To Find Taper Per Foot

In duplicating a taper the gage jaws may be set to the model and by placing between the jaws a pair of disks whose diameters are known the taper per foot may be readily found. For example, the jaws in Fig. 4 are set to a certain model, two disks 0.9 and 1.1 inch diameter are placed between them and the distance over the disks measured, from which dimension l (which is 3.5 inches) is readily found by subtracting half the diameters of the disks. Here l represents the center distance as in Fig. 2. To determine the taper per foot which may be represented by T, the formula is:

$$T = 24 \left(\frac{R-r}{\sqrt{l^2 - (R-r)^2}} \right).$$

Then

$$T = 24 \left(\frac{0.1}{\sqrt{12.25 - 0.01}} \right) = 24 \left(\frac{0.1}{3.4985} \right) = 0.684$$

Taper per foot = 0.684 inch.

To Find Width of Opening at Ends

If, with the ends of the gage jaws flush with a line tangent to the disk peripheries as in Fig. 5, it is required to find the width of the opening at the small end where a represents one half that width, the following formula may be applied, the disks being as in the last example 0.9 and 1.1 inch diameter respectively, and the center distance 3.5 inches:

$$a = r\sqrt{\frac{l - (R - r)}{l + (R - r)}}$$

Then

$$a = 0.45 \sqrt{\frac{3.5 - (.55 - .45)}{3.5 + (.55 - .45)}}$$

$$= 0.45 \sqrt{\frac{3.4}{3.6}} = 0.45 \sqrt{.94444} = .4373$$

0.4373 inch \times 2 = 0.8746 inch width of opening at small end of gage.

Similarly the width of opening at the large end of the gage may be found as follows, where b = half the width of the large end.

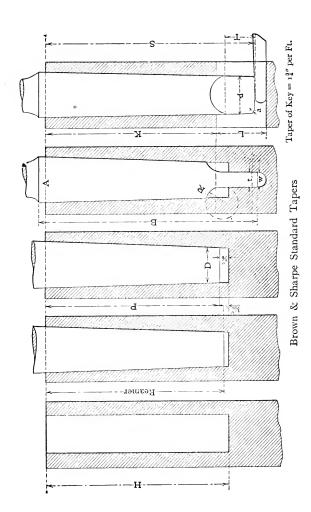
$$b = R\sqrt{\frac{l + (R - r)}{l - (R - r)}}$$

Then

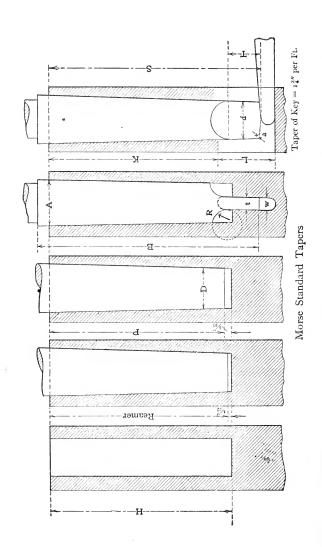
$$b = 0.55 \sqrt{\frac{3.5 + (.55 - .45)}{3.5 - (.55 - 45)}}$$

$$= 0.55 \sqrt{\frac{3.6}{3.4}} = 0.55 - 1.05882 = .56595$$

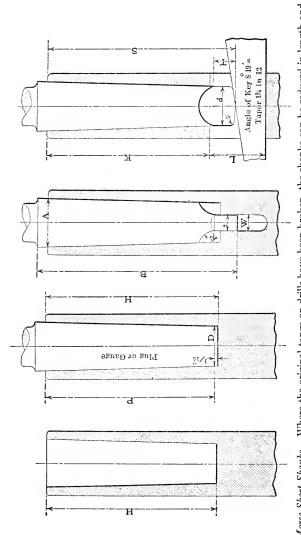
0.56595 inch \times 2 = 1.1319 inch = width of opening at large end.



Taper per Inch		0416 0416 0416 0416 0416 0416 0416 0416
Taper per		500 500 500 500 500 500 500 500
Shank Depth	s	н н н н и и и 4 4 4 мо г го о о о н « — инмани — инманительной они — и — и и и о о о о о о о о о о о о о
Radius of	g	0.00 0.00 0.00 0.00 0.00 0.00 0.00 0.0
Radius of Mill for Tongue	R	minute in the common control of the
Thickness of Tongue	-	ಗ∞್ಯಾಪ್ ಸ್ಟ್ರ್ ಗ ಪ್ರಾಣ್ಣ ಗ ಸ್ಟ್ರಾಪ್ ಸ್ಟ್ರಾಪ್ ಸ್ಟ್ರಾಪ್ ಸ್ಟ್ರ್ ಸ್ಟ್ರಿಸ್ಟ್ ಸ್ಟ್ರಾಪ್ ಸ್ಟ್ರಾಪ್ ಸ್ಟ್ರಾಪ್ ಸ್ಟ್ರಾಪ್ ಸ್ಟ್ರಿಸ್ಟ್ ಸ್ಟ್ರಾಪ್ ಸ್ಟ್ರಾಪ್ ಸ್ಟ್ರಾಪ್ ಸ್ಟ್ರಿಸ್ಟ್ ಸ್ಟ್ರಾಪ್ ಸ್ಟ್ರಿಸ್ಟ್ ಸ್ಟ್ರಾಪ್ ಸ್ಟ್ರಿಸ್ಟ್ ಸ್ಟ್ರಾಪ್ ಸ್ಟ್ರಿಸ್ಟ್ ಸ್ಟ್ಟ್ ಸ್ಟ್ಟ್ರಿಸ್ಟ್ ಸ್ಟ್ಟ್ಟ್ ಸ್ಟ್ಟ್ಟ್ ಸ್ಟ್ಟ್ಟ್ಟ್ ಸ್ಟ್ಟ್ಟ್ಟ
Dia. of Tongue	р	2170 280 280 380 380 360 560 560 1010 1010 1010 1010 1010 1010
Length of Tongue	T	alphanphidaphona, daphonosiasiasiasiasiasiasiasiasiasiasiasiasias
Width of Key-	W	7.61 - 7.62 - 7.
Length of Keyway	L L	CONTRACTOR THE
End of Socket to Keyway	K	Н Н Н И И И И И Д 4 И И О С КО О О Н Н Н И И И И И Д 4 И И О О КО О О Н Н Н И И И И И О Д 4 И И О О О О Н Н Н И И И И И О О О О О О О О О О О О
Depth of Hole	Н	H
Whole Length of Shank	В	1 1 1 1 2 2 2 4 4 200 1 1 2 2 2 1 1 1 1 1 2 2 2 2 4 4 200 1 1 1 1 2 2 2 2 2 2 2 2 2 2 2 2 2 2
Standard Plug Depth	Ъ	H
Dia. at End of Socket for length P.	A	2301 22955 39552 39552 5529 5989 77250 10060 1.2596 1.2596 1.2596 1.2596 1.2596 1.2596 2.3437 2.3437 2.3437 2.8855
Dia. of Plug at Small End	D	25.5 3.12 3.12 3.12 3.12 3.00 9.00 1.0446 1.50
Taper of		H 2 2 4 2 0 0 0 0 1 1 2 2 4 4 1 1 1 1 1 1 1 1 1 1 1 1 1 1

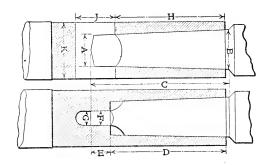


No. of Key		0	н	а	m	4	ν	9	7
Taper per Inch		.05208	.05	91050.	91050.	.0519F	.0525	.05216	.05208
Taper per Foot		.625	009.	.602	209.	.623	.630	.626	.625
Shank Depth	s	2) 1,00 101	$2\frac{7}{16}$	$\begin{array}{c}2_{15}^{15}\end{array}$	$3\frac{11}{16}$	2 t s	70 F.8	8.1.4.	114
PagaoT To zuibaЯ	a	ó. 4	.05	90.	80.	01.	.12	.15	81.
Radius of Mill for Tongue	. 🛪	10 01 01	es ju	-4	on job	7 N	miss	⊷ic1	60 /4
Thickness of Tongue	t t	10 02 0.3	m 120	⊢ (**	1.5	10,51 ⊷100	10/20	col-st	I \$
Dia. of Tongue	Ф	.235	.343	C 03	62/62 62/63	00 00 14 01	H (50)	61	C3 co/co
Length of Tongue	H	⊢ (4	co (∞	16	16	ratoo	60 44	H ₈	H 8/30
Width of Key-	W	091.	.213	.26	.322	.478	.635	* 94.	1.135
Length of Key-	ы	o.ho	65 -4	r- 20	I To	1.4.	I 25	H ⊗ 44	01 10 10
End of Socket to	×	115	$2\frac{1}{16}$	231	$3\frac{1}{16}$	334	415	7	\$6
Depth of Hole	H	$2\frac{1}{32}$	$2\frac{3}{16}$	C) rcixo	3.1	48	54	788	1018
Whole Length of Shank	В		$2\frac{9}{16}$	3 1	3.7	4 8 4 7	68	818	I I §
Standard Plug Depth	ы	63	2 8 1	2 1 6	318	4 <u>1</u> 6	518	74	01
Dia. at End of Socket	A	.356	.475	7.	.938	1.231	1.748	2.494	3.27
Dia. of Plug at Small End	Q	.252	.369	.572	.778	1.02	1.475	2.116	2.75
TageT to .oV		0	I	6	co	4	25	9	7



provided with thicker and wider tangs for insuring a stronger drive, the shanks being fitted to a tang gage for short shank sockets. The short shank drills are then used in sleeves and sockets which are made externally to fit the Morse Short Shanks. Where the original tangs on drills have been broken, the shanks can be reduced in length and regular Morse socket and spindle holes.

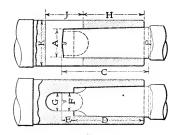
	plet of Key	un _N	0	н	64	3	4	S	9	7
	ст рет Ілсһ	qsT	.05208	.050	91050.	91050.	16150	0525	.05216	.05208
	er Der Foot	дьТ	.625	900	.602	.602	.623	.630	.626	.625
	End of Socket	×	F 1 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	132	13.7 2.8 1	235	23.1	333	513	-1 8
vay	Length of	1	ue) ao	133	113	116	1 2	7	C1 60 41	S⊗
Keyway	Width of Keyway	M	.193	.269	3855	512	.037 149.	1.016	1.268	1.644
	Radius of Tongue	в	8 ¹ 29	18	$\frac{1}{16}$	eg jes	3,	r(x)	riso	18
	Diameter of Tongue	p	.258	.371	.575	.783	1.023	1.483	2.128	2.769
Tongue	Rad. of Mill for Tongue	×	I e	H4	a (c)	n m	m(10	40	NO 00	m +
	Length of Tongue	T	-4•	5 to 10	7 E	9.1 9.1	NO (OR	⊘ 4	⊢(s)	H 24
	Thickness of Tongue	+	186	.251	.374	.50I	.626	100.1	1.251	1.627
	Standard Plug Depth	Ъ	10/100 H	H 614	61	$21_{\bar{6}}^{7}$	3 4	4 8	S 85	718
	Depth of Hole	н	1831	113	$2\frac{1}{16}$	7 7	318	41^{3}_{6}	η. 6)4	$8_{\mathrm{I}^{3}}$
nk	Shank Depth	S	H (4.5	61	CI Wino	215	313	413	6 3	916
Shank	Whole Length .	В	H €8 100	$2\overline{1}\frac{3}{6}$	21 ⁹	s	413	51^{3}	713	943
cket	Diam, at End of So	A	.356	.475	.70	.938	1.231	1.748	2.494	3.270
	Diam. of Plug at Small End	D	172.	.388	009.	918.	1.062	1.532	2.201	2.857
	Number of Taper		٥	н	(1	٠ د	4	Ŋ	9	7



THE STANDARD TOOL COMPANY'S STANDARD TAPER SHANKS

No. of Taper	Diameter Small End of Shank	Diameter Large End of Shank	Total Length of Shank	Depth Hole in Socket	Length Tongue to End Socket Hole	Thickness of Tongue
0 1 2 3 4 5 6	A .2406 .3533 .5531 .7529 .9908 I.4390 2.0638 2.6849	B .3626 .4814 .7099 .9472 1.2438 1.7605 2.5104 3.2903	$\begin{array}{c} C\\ 2\frac{1}{3}\frac{1}{3}\frac{1}{2}\\ 2\frac{1}{16}\frac{1}{6}\\ 3\frac{1}{8}\\ 3\frac{1}{8}\\ 4\frac{1}{8}\\ 6\frac{1}{8}\\ 8\frac{1}{16}\\ 11\frac{1}{8}\\ \end{array}$	D 2 3 5 6 2 5 1 4 1 6 1 4 1 6 1 7 7 8 1 0 8	E. 316.515.515.75.62.518.78.18.18	F 53354 4 5652 8254 18

No. of Taper	Width of Keyway	End of Socket of Keyway	Length of Keyway	Diameter of Socket	Taper per Foot	Taper per Inch
0 1 2 3 4 5 6	G 144 78274414-14-14-694 94	$\begin{array}{c} H \\ 1\frac{1}{1}\frac{5}{16} \\ 2\frac{1}{16} \\ 2\frac{1}{2} \\ 3\frac{1}{16} \\ 3\frac{1}{8} \\ 4\frac{1}{16} \\ 7 \\ 9\frac{1}{2} \end{array}$	J 9 16 0 14 16 16 17 17 19 14 14 15 16 17 17 19 14 14 15 16	$\begin{array}{c} : \ K \\ \frac{9}{100} \frac{1}{200} \frac{1}{200} \\ \frac{1}{200} \frac{1}{200} \\ \frac{1}{200} \frac{1}{200} \frac{1}{200} \\ \frac{1}{200}$.625 .600 .602 .602 .623 .630 .626	.05208 .05000 .05016 .05016 .05191 .05250 .05216



THE STANDARD TOOL COMPANY'S SHORT TAPER

No. of Taper	Diameter Small End of Shank	Diameter Large End of Shank	Total Length of Shank	Depth Hole in Socket	Length Tongue to End Socket Hole	Thickness of Tongue
	A	В	C	D	E	F
I	.378	.484 .706 ·	$2\frac{1}{8}$	$1\frac{3}{4}$	$\frac{1}{4}$	1/4
2	.378 .587	.706	$2\frac{1}{8}$ $2\frac{3}{8}$	I 15	$\frac{5}{16}$	3/8
3	.800	.941	$2\frac{1}{1}\frac{3}{6}$	$2\frac{1}{4}$	1 6 3 8	$\frac{1}{2}$
4	1.050	1.244	$3\frac{3}{4}$	3	16	<u>5</u> 8
5 6	1.515	1.757	34 45 63 63 8	$3\frac{3}{4}$	9 16 55 83 4	I
6	2.169	2.501	$6\frac{3}{8}$	5	3/4	I 1/4
7	2.815	3.283	9	$7\frac{3}{4}$	I	$1\frac{1}{2}$

No. of Taper	Width of Keyway	End of Socket to Keyway	Length of Keyway	Diameter of Socket	Taper per Foot	Taper per Inch
	G	Н	T	K		
I	.263	$1\frac{5}{8}$ $1\frac{3}{4}$	34	$\frac{2}{3}\frac{5}{2}$.600	.0500
2	.388	$1\frac{3}{4}$	I	I 1 6	.602	.05016
3	.520	2	$1\frac{1}{4}$	1 5 1 6	.602	.05016
4	.645	$2\frac{1}{16}$	I 1/2	$1\frac{1}{1}\frac{1}{6}$.623	.05191
5	1.020	$3\frac{1}{4}$ $4\frac{3}{8}$	2	$2\frac{7}{16}$.630	.0525
6	1.270	$4\frac{3}{8}$	$2\frac{1}{2}$	2 7/8	.626	.05216
7	1.520	7	3		.625	.05208

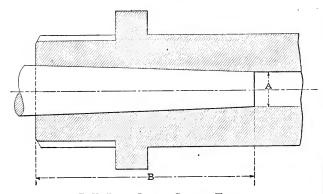
THE STANDARD TOOL CO'S. SHORT TAPER SHANKS

THE table shows the short taper shanks of the Standard Tool Company for giving a tang of increased strength.

Sockets and sleeves are furnished, made with the outside taper to fit the regular taper of spindles of drill presses; the inner taper being suitable for the short shanks and also made with both outside and inside taper, conforming to the new standard, and these latter interchange or nest into each other.

THE REED TAPER

The F. E. Reed Company, Worcester, Mass., uses in its lathe spindles the 1 in 20 taper (0.6 per foot) which the Jarno system is based on. The diameters of the Reed tapers, however, differ from the Jarno, and the lengths in most cases are somewhat less. The dimensions are given in the table below.



F. E. REED LATHE CENTER TAPERS

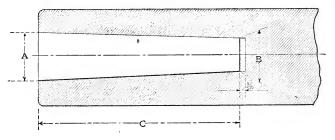
TAPER PER FOOT = 0.6 INCH. TAPER PER INCH = 0.05 INCH

Size of	Dia. of Small End of Taper	Length of Taper	Size of	Dia. of Small End of Taper	Length of Taper
Lathe	A	В	Lathe	A	В
12" 14" 16" Special 16" 18"	9 1556 143 144 144	$ 3\frac{5}{8} $ $ 4\frac{8}{8} $ $ 4\frac{8}{8} $ $ 4\frac{3}{8} $ $ 5\frac{1}{16} $	20" 22" 24" 27" 30"	$ \begin{array}{c} 1\frac{1}{2} \\ 1\frac{1}{2} \\ 1\frac{3}{4} \\ 1\frac{3}{4} \\ 2 \end{array} $	5 1 5 6 5 1 2 2 5 2 3 3 4

THE JARNO TAPER

While the majority of American tool builders use the Brown & Sharpe taper in their milling-machine spindles and the Morse taper in their lathes, a number of firms, among them the Pratt & Whitney Company, Hartford, Conn., and the Norton Grinding Company, Worcester, Mass., have adopted the "Jarno" taper, the proportions of which are given in the accompanying table. In this system the taper of which is 0.6 inch per foot or 1 in 20, the number of the taper is the key by which all the dimensions are immediately deter-

mined without the necessity even of referring to the table. That is, the number of the taper is the number of tenths of an inch in diameter at the small end, the number of eighths of an inch at the large end, and the number of halves of an inch in length or depth. For example: the No. 6 taper is six eighths $(\frac{3}{4})$ inch diameter at large end, six tenths $(\frac{6}{10})$ diameter at the small end and six halves (3 inches) in length. Similarly, the No. 16 taper is $\frac{16}{5}$, or 2 inches diameter at the large end; $\frac{16}{10}$ or 1.6 inches at the small end; $\frac{16}{2}$ or 8 inches in length.



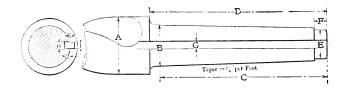
JARNO TAPERS

Taper per foot = 0.6 inch. Taper per inch = 0.05 inch.

Dia. Large End =
$$\frac{\text{No. of Taper}}{8}$$

Dia. Small End = $\frac{\text{No. of Taper}}{10}$
Length of Taper = $\frac{\text{No. of Taper}}{2}$

No.	Dia. Large End	Dia. Small End	Length of Taper	No.	Dia. Large End	Dia. Small End	Length of Taper
Taper	A	В	С	Taper	A	В	С
1 2 3 4 5 6 7 8	.125 .250 .375 .500 .625 .750 .875 1.000	.10 .20 .30 .40 .50 .60 .70 .80	.5 1. 1.5 2.0 2.5 3.0 3.5 4.0 4.5	11 12 13 14 15 16 17 18	1.375 1.500 1.625 1.750 1.875 2.000 2.125 2.250 2.375	1.10 1.20 1.30 1.40 1.50 1.60 1.70 1.80	5.5 6.0 6.5 7.0 7.5 8.0 8.5 9.0
10	1.250	1.00	5.0	20	2.500	2.00	10.0



SELLERS TAPERS

A B C D E F G H												
	Width of Key Hight of Key	Depth of Spline in Shank	Width of Spline in Shank	Approximate Pitch of Spiral Grooves		Dia. at Reduced Portion of Shank	Length of Shank Over All	Total Length of Drill	Length of Drill Body from Point B	Length of Shank from Point B		Dia. of Drill
$\begin{array}{c ccccccccccccccccccccccccccccccccccc$	I K	Н	G		F	E	D			С	В	A
$\begin{array}{cccccccccccccccccccccccccccccccccccc$	5 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0	44 44 44 44 44 44 44 44 44 44 44 44 44	1	6.24 6.24 7.28 7.28 9.50 """"""""""""""""""""""""""""""""""""	66 66 66 66 66 66 66 66 66 66 66 66 66	1 1 2 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	27	10 11 ¹ / ₂ 12 12 13 ¹ / ₂ 13 ¹ / ₂ 13 ¹ / ₂ 13 ¹ / ₂ 14 14 ¹ / ₂ 14 ¹ / ₂ 16 ¹ / ₂ 16 ¹ / ₂	9 9 9 9 9 12 10 10 10 10 10 10 10 10 11 11	$\begin{array}{c} {}_{1} \\ {}_{2} \\ {}_{3} \\ {}_{4} \\ {}_{4} \\ {}_{6} \\ {}_{6} \\ {}_{6} \\ {}_{7} \\ {}_{1} \\ {}_{1} \\ {}_{2} \\ {}_{3} \\ {}_{1} \\ {}_{2} \\ {}_{3} \\ {}_{4} \\ {}_{6} \\ {}_{7} \\ {}_{1} \\ {}_{2} \\ {}_{3} \\ {}_{4} \\ {}_{6} \\ {}_{6} \\ {}_{7} \\ {}_{1} \\ {}_{2} \\ {}_{3} \\ {}_{4} \\ {}_{6} \\ {}_{6} \\ {}_{7} \\ {}_{6} \\ {}_{7} \\ {}_{6} \\ {}_{7} \\ {}_{7} \\ {}_{6} \\ {}_{7} \\ {}_{7} \\ {}_{8} \\ {}_{7} \\ {}_{8} \\ {}_{7} \\ {}_{8$	11 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	14 5 5 3 8 6 1 7 1 1 2 2 7 1 5 8 1 1 7 3 4 2 1 7 1 4 5 1 6 6 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1

THE SELLERS' TAPER

The system of tapers used by William Sellers & Company, Inc., of Philadelphia, Pa., in lathes, drilling and boring machines, is given in the preceding table. The taper is $\frac{3}{4}$ inch per foot and each size of taper is splined as shown for a key the dimensions of which are included in the table. The pitch of the spiral for the drills used by the company is also included.

TAPER PINS AND REAMERS



TAPER REAMERS AND PINS (PRATT & WHITNEY CO.)

Taper = $\frac{1}{4}$ inch per foot or .0208 inch per inch

Size. No.	Dia. of Small End of Reamer	Dia. of Large End of Reamer	Length of Flute	Total Length of Reamer .	Size Drill for Reamer	Longest Limit Length of Pin	Dia. of Large End of Pin	Approx. Fractional Size at Large End of Pin
0 1 2 3 4 5 6 7 8 9 10 11 12 13	0.135" .146" .162" .183" .208" .240" .279" .331" .398" .482" .581" .706" .842"	.162" .179" .200" .226" .257" .300" .354" .423" .507" .609" .727" .878" I.050"	15/8" 1 1/8" 1 1/8" 1 1/8" 2 1/8" 2 1/8" 3 18" 4 1/8" 6 8" 7" 8 14" 1 2"	2" 2 \$ 1 1 6" 2 \$ 1 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6 6	28 25 19 12 3 14 951100014400200014 100000056 11 16	1" 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	.156" .172" .193" .219" .250" .289" .341" .409" .706" .857" 1.013"	5/2/1/4 // 5/4/1/5/4 // 5/4/1/5/4 // 5/4/1/5/4 // 5/4/1/5/4 // 5/4/1/5/4 // 5/4/1/5/4 // 5/4/1/5/4 // 5/4/1/5/4

These reamer sizes are so proportioned that each overlaps the size smaller about $\frac{1}{2}$ inch.

TABLE OF DRILL SIZES FOR TAPER PINS

		TAB	LE OF I	RILL	SIZES	FOR	TAPER P	INS	
			Taper Pir		No. of Pin	L	D	d	No. of Drill
taper	pins r	anging i	n lengths	$\text{by}_{\frac{1}{4}}$		- 1"	2 122"	0 //	25#
			ch long, t		7	I 1/4" I 1/2" I 3/4"	0.409" 0.409"	0.3829"	25" 8 4" 3" 3"
10, 6	inches	long.	The diame	ter of	7 .	1 2 7 3"	0.400″	0.3777"	3//
the small end of the pin for each						2 "	0.409"	0.3725" 0.3673"	8 3 //
length is given in the fourth column with the drill size in the fifth column.						21"	0.409"	0.3621"	
with t	he dril	l size in t	he fifth co	lumn.	7 7	2 1/1 2 1/2 2 1/2 2 3/1 2 3/1	0.409"	0.3569"	8/11 8/21 8/21 8/21 8/21 8/21 8/21 8/21
					7	23"	0.409"	0.3517"	23//
						1 2 "	0.409"	0.3465"	11/
-	T T			.	7	31/	0.409"	0.3413"	11"
	DII .) d	7 7 7	3 ½" 3 ½" 3 ½" 3 ½" 1 ½"	0.400"	0.3361"	11"
	¥.\			-4-	7	3 3"	0.409"	0.3300"	21"
	H	I			8	I 1"	0.492"	0.466 "	35"
)	d=D-	Length of Pin	X .25	8	T 1"	0.492"	0.4608"	35"
	9	4-D-	12		8	I 3/4	0.492"	0.4555"	29/
	1				8		0.492"	0.4503"	84 29#
					8	2 1/1 2 1/1 2 1/1	0.492"	0.4451"	84
					8	2 3"	0.192"	0.4399″	18"
	1	l .	1	1 27 6	8		0.492"	0.4347"	18
No.	L	D	d	No. of	0		0.492"	0.4295"	16
Pin		-		Drill	0	3½" 3½"	0.492" 0.492"	0.4243"	16 27" 84 27"
					8	3 2"	0.492"	0.4139"	
•	3"	0.156"	0.1404"	28	7 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8 8	34"	0.492"	. 0.4087"	6 4 1 3/1
0	3" 1"	0.156"	0.1352"	20	8	11"	0.492"	0.4035"	132"
1	3"	0.172"	0.1564"	22	8	41/2"	0.492"	0.3982"	132"
ī	т ″	0.172"	0.1512"	24	9	T 1/2"	0.591"	0.5597"	9"
ī	I 1/4 / 3/4 / 4	0.172"	0.146 "	26	9	I 3"	0.591"	0.5545"	35"
2	3//	0.193"	0.1774"	16	9		0.591"	0.5493"	35/
2	I "	0.702"	0.1722"	17	ģ	21"	0.591"	0.5441"	3 5"
2	11/1	O TO2"	0.167 "	18	9	25"	0.501"	0.5380"	35"
2	$\begin{array}{c} I\frac{1}{4}'' \\ I\frac{1}{2}'' \end{array}$	0.193	0.1618"	20	9	2 4"	0.591"	0.5337"	37"
3	3"	0.219	0.2034"	6	9	3 "	0.591"	0.5285"	17" 32"
3	1"	0.219"	0.1982"	8	9	31"	0.591"	0.5233"	32"
3	I 1 1 " I 1 2 "	0.219"	0.193 "	10	9	32"	0.591"	0.5181"	64
3 3 3 4	I ½"	0.219"	0.1878"	12	9	3½" 3½" 3¼"	0.591"	0.5129"	33" 64 33"
3	I 3/4	0.219"	0.1825"	14	9	11/1	0.591"	0.5077"	61/
4		0.250"	0.2344"	15#	9	4 ¹ / ₄ " 4 ¹ / ₂ "	0.591" 0.591"	0.5025" 0.4972"	1/2
4	I "	0.250"	0.2292	I 64	9	42	0.591"	0.4972	1/2"
4	I 1/4" I 1/2"	0.250"	0.2187"	2	9	4 ³ / ₄ "	0.591"	0.4868"	31/
4	T 3"	0.250"	0.2135"	3	9	51"	0.591"	0.4816"	31"
4	2""	0.250"	0.2083"	1	l ó	-1"	0.591"	0.4764"	31"
455555556	211	0.289"	0.2734"	17" 64 17" 64 17" 64	10	13"	0.706"	0.6747"	43"
5	I "	0.289"	0.2682"	17/	10	I 3/"	0.706"	0.6695"	43"
- 5	τ1"	0.289"	0.263 "	64"	10	2 "	0.706"	0.6643"	43/
5	1 ½" 1 ½" 2 "	0.289"	0.2577"	1"	10	21"	0.706"	0.6591"	32
5	I 3"	0.289"	0 2525"	1/1/	10	2 ½" 2 ¾"	0.706"	0.6539"	32
5		0.289"	0.2473"		10	24"	0.706"	0.6487"	32 41"
5	2 1 " 3 "	0.289"	0.2421	15" 64 21" 84	10		0.706"	0.6435"	41" 64 41"
6		0.341"	0.3254"	21" 84 21"	10	3 ½" 3 ½"	0.706"	0.6383"	64
6	1 1	0.341"	0.3201"	5 / / I d	10	32	0.706"	0.6331" 0.6278"	64
6 6	I 1/4" I 1/2"	0.341"	0.315 "	18	10	34"	0.706"	0.6276"	5/1
6	I 3/4"	0.341"	0.310	18 19" 84"	10	41/	0.700"	0.6174"	5//
6	2"	0.341"	0.3045"	19#	10	44"	0.706"	0.6122"	39"
6	21"	0.341"	0.2994	19/1 64 19/1 64	10	4½" 4¾"	0.706"	0.6078"	39#
6	21"	0.341"	0.2889"	9.//	10	- "	0.706"	0.6018"	39"
6	23/1	0.341"	0.2837"	32 9 #	10	51"	0.706"	0.5966"	19/
6	1 2 "	0.341"	0.2785"	32 #	10	51/2"	0.706"	0.5014"	32
6	31"	0.341"	0.2733"	32 17" 64 25"	10	53"	0.706"	0.5862"	32 19# 32 37#
7	T*"	0.400"	0.3881"	25"	10	54"	0.706"	0.581 "	37#

STANDARD TAPER PINS USED BY THE U.S. ORDNANCE DEPARTMENT

THE accompanying tables show the standard taper and split pin used by the Ordnance Department of the U.S. Army as revised up to July 20, 72. The first table shows the standard taper pins and the retaining split pins used in the end. The pins are \$\frac{1}{2}\$-inch taper per foot. They have tapered head and point to allow for upsetting, either in driving or removing pins from the work.

The first table gives the number of the pin and the diameter at the small end in the first two columns. The next three columns show the sizes

of the taper pin to prevent accidental withdrawal from the hole.

of the shaft and collars in which these pins are used, while the last two columns show the split pins which are used in some cases on the small end The second table shows the dimensions of the standard split pin used in connection with the taper pins, and gives the diameter and the length manufactured. The tapered points are shown by the drawing to be the length of the diameter of the split pin, with the small end of the taper 0.75

STANDARD TAPER PINS																					
STANDARD TAPER PINS		T									4	4	4	4							
STANDARD TAPER PINS						9					3.75	3.75	3.75	3.75							
STANDARD TAPER PINS		1				S.L.	ZES				3.5	3.5	3.5	3.5							
STANDARD TAPER PINS						THI	ALS				3.25	3.25	3.25	3.25							
STANDARD TAPER PINS STANDARD SPLIT FINS						TOW	ERCI					es									
STANDARD TAPER PINS STANDARD SPLIT FINS			Ì			SBE	MMC			2.75	2.75	2.75	2.75		0						
STANDARD TAPER PINS STANDARD SPLIT FINS						FIN	E OC	2.5	2.5	2.5	2.5	2.5	5.5		75						
STANDARD TAPER PINS STANDARD SPLIT PINS	S	1				ALI	ΥI	2.25	2.25	2.25	2.22	2.25	2.25		-1°H						
STANDARD TAPERPINS and Retaining Split Pins USEDIN SHAPTS USEDI	l PI		ED				21		C1						Tha						
STANDARD TAPERPINS and Retaining Split Pins USEDIN SHAPTS USEDI	LIJ		CLC				1.75	1.75	1.75	1.75	1.75	1.75			th th						
STANDARD TAPERPINS and Retaining Split Pins USEDIN SHAPTS USEDI	SI		UFA				1.5	1.5	1.5	1.5	1.5	1.5			a Tha						
STANDARD TAPERPINS and Retaining Split Pins USEDIN SHAPTS USEDI	ARI		MAN			,	1.25	1.25	1.25	1.25	1.25										
STANDARD TAPERPINS and Retaining Split Pins USEDIN SHAPTS USEDI	ND,		THS			55									(0)						
STANDARD TAPERPINS and Retaining Split Pins USEDIN SHAPTS USEDI	šTA	1	ENG		0.75	0.81	-	_	_	-	_		h	ű	1.0r						
STANDARD TAPER PINS	02	1	-		5						-	-	6 inc	6 inc	Diam fole in						
STANDARD TAPER PINS				0.5	0.562	0.625	0.75	0.75	0.75	0.75			and	and	nal I of II ve P						
STANDARD TAPER PINS					-	-	-	_	_	-		-	805	80 5	omin lam.						
STANDARD TAPER PINS				315	375	437	.5	.5	5				Al	Ā	ÄÜÄ						
STANDARD TAPER PINS and Retaining Split Pins NORK USEDIN SHAPTS GOLLARS DIAM. LENGTH Mar. Diam. Mar. Diam. 0.3 0.75 0.3 0.75 0.5 0.75 1125 0.0463 0.3125 0.5 0.75 1126 0.0463 0.375 1.5 2 3.25 0.0437 0.475 1.5 2 3.25 0.0437 0.475 1.5 2 3.25 0.0437 0.475 1.5 2 0.027 0.126 1.5 0.25 0.125 1.		-	+	\vdash	+	-	-	-	-	+-	0.3	-	-	-	_		-	-		Н	Ω
STANDARD TAPER PINS and Retaining Split Pins NORK USEDINSHAFTS COLLARS DIAM. LENGTH Min.Diam. Mar.Diam. 0.3 0.75 0.3 0.75 0.5 0.75 11.25 0.0463 0.375 0.5 0.75 11.25 0.0463 0.375 1.5 2 3.25 0.06375 0.5 1.5 2 3.25 0.06375 0.5 1.5 2 4.5 0.025 0.125 1.5 2 0.025 0.125 1.5 2 0.025 0.125 1.5 2 0.025 0.125 1.5 2 0.025 0.125 1.5 2 0.025 0.125 1.5 0.5 0.025 1.5 0.02											0468	0463	.0463	.0463	625	781	9375	55	56	93	2
STANDARD TAPER PINS WORK WAS Dian. WAS DIA			<u> </u>	0	0.	3	0.0	0	0	0	9	0	0	9	цg						
STANDARD TAPER PINS WORK WAS Dian. WAS DIA															hrou						
STANDARD TAPER PINS WORK WORK WORK WORK WORK WORK WORK WORK WALDIAN MANDIAN WALDIAN	S		тн				52	2	22		Γ			in t							
STANDARD TAPER PINS WORK WORK WORK WORK WORK WORK WORK WORK WALDIAN MANDIAN WALDIAN Ė	3	INS	ENG				0.31	0.37	0.43	0.5	0.75		1.25	lit I							
STANDARD TAPER PI WORN USEDIN SHAPTS GOLIA Min.Piam. Mar.Diam, Mar.Diam O.5 0.75 1.02 O.5 0.75 1.102 O.5 0.75 1.102 O.5 0.75 1.103 O.5 0.75 1.103 O.5 0.75 1.02 O.5 0.75 0.02 O.5 0.75 0.02 O.7 0.02 O.7 0.02 O.7 0.03 O.7 0.0	:	Spin	II P	-		-	-	_	_	-	20	-	-	-	2 → 1 - 1						
STANDARD TAPER PI WORN USEDIN SHAPTS GOLIA Min.Piam. Mar.Diam, Mar.Diam O.5 0.75 1.02 O.5 0.75 1.102 O.5 0.75 1.102 O.5 0.75 1.103 O.5 0.75 1.103 O.5 0.75 1.02 O.5 0.75 0.02 O.5 0.75 0.02 O.7 0.02 O.7 0.02 O.7 0.03 O.7 0.0		1ng	SPI	IAM.				0463	0625	0781	0937	125	156	203							
STANDARD TAPER PI WORN USEDIN SHAPTS GOLIA Min.Piam. Mar.Diam, Mar.Diam O.5 0.75 1.02 O.5 0.75 1.102 O.5 0.75 1.102 O.5 0.75 1.103 O.5 0.75 1.103 O.5 0.75 1.02 O.5 0.75 0.02 O.5 0.75 0.02 O.7 0.02 O.7 0.02 O.7 0.03 O.7 0.0		Tail						0	0	9	0	o,	9	0	4						
STANDARD TAPES wound wound Wound USEDIX SHAFTS GA Min. Diam. Max. Diam. Ma 0.3 0.3 0.5 1.5 1.5 2 3 4.5 4.5 4.5 0.5 D Minches A Minches	INE	Fe		ARS	Jiam.	2		25	52	35	2	15	23	22	(())						
STANDARD TAPP STANDARD STANDAR	R P			COLL	dax.I	0.7		1.1	1.6	2.15	3.2	4.8	9.9	9.6							
STANDARD T and TARER PINS World TARER PINS WORLD MAILEND USEDIN SHAFTS Number D Min-Diam. Mar.Dia 00025 0 0.005 0.	APE		,,		m. N	-	-	-	-	-	-	-	-		1						
STANDAR STANDAR STANDAR STANDAR STANDAR STANDAR STANDAR STANDER STAN	D T.	and	YORE	FTS	x.Dia	0.3	0.5	0.75	_	1.5	c1	3	4.5	6.5	a 2/10=						
STAND STAN	ARI	-	-	SHA	Ma				L	L	L	_									
A	ND	-		DIN	Diam.		63	5	7.5		22			25							
0.25 per Ft. Taper TAPER PINS DIA. AT SMALL END Number D 0 0.025 0 0.025 2 0.0162 2 4 0.208 6 0.279 8 0.308 10 0.551 12 0.452 PARTER PINS D 0.551	STA			USE	Min.		0	0	0	-	7	2	3	-14	to ()						
0.25 per Ft. Ta, TAPER PINS DIA. AT SMALL E. Number 0.002 0.0135 2 0.0185 2 0.0185 2 0.0279 10 0.279 10 0.581		ig.		a.		10	7.2					_									
0.25 per Ft TAPER P DIA. AT SMA Number 0 0 2 4 6 10 12	E	a l	INS	IL E	Ω	0.062	0 003	0.135	0.162	3.208	0.279	0.398	0.581	0.842	7						
0.25 pe	i	FE	ER P	SMA	-	L	_	-	Ľ	F	F	-	F	_	'						
10 0 0.1	,	50 pe	TAP	AT.	ampe			0		4		80		12							
		=		νīα	ž			L	C1	Ļ	9		2	L							

Tapers from $\frac{1}{16}$ to $1\frac{1}{4}$ inch per Foot amount of taper for lengths up to 24 inches

th of Portion	Taper per Foot													
Leng Tapered	18	32	18	1	38	1/2	58	3	I	114				
Longth of Long and Longth of Longtho	.0002 .0003 .0010 .0013 .0016 .0020 .0023 .0026 .0033 .0036 .0039 .0049 .0052 .0104 .0156 .0208 .0210 .0210 .0210 .0252 .0210 .0252 .0210 .0252 .0210 .0252 .0256 .0257	.0002 .0005 .0010 .0024 .0029 .0034 .0059 .0054 .0059 .0068 .0073 .0078 .0312 .0391 .0469 .0547 .0625 .0703 .0781 .0859 .0937 .0156	.0003 .0007 .0013 .0020 .0026 .0039 .0046 .0052 .0059 .0072 .0078 .0104 .0208 .0312 .0729 .0833 .0937 .0142 .1146 .1125	.coo7 .coo7 .coo39 .coo52 .coo65 .coo91 .cloo4 .cloo7 .cloo2 .cloo2 .cloo2 .cloo2 .cloo2 .cloo2 .cloo2 .cloo2 .cloo3 .cloo2 .cloo3 .clo	.0010 .0020 .0039 .0078 .0098 .0117 .0137 .0156 .0176 .0214 .0224 .0234 .0234 .0234 .0213 .0312 .0937 .1252 .11562 .2187			.0020 .0039 .0078 .0117 .0156 .0234 .0273 .0312 .0352 .0391 .0430 .0469 .0508	.0026 .0052 .0104 .0156 .0208 .0260 .0312 .0365 .0417 .0573 .0625 .0677 .0729 .0781 .0833 .1667 .250 .3333 .4167 .500 .5833 .9167 .750 .8333 .9167	.0033 .0065 .0130 .0195 .0260 .0326 .0391 .0456 .0521 .0716 .0781 .0846 .0911 .0977 .1042 .2083 .3125 .4167 .5208 .625 .7292 .8333 .937 .10417 1.1458 1.250 1.3542 1.4583 1.5625				
16 17 18 19 20 21 22 23 24	.0937 .0990 .1042 .1094 .1146	.1328 .1406 .1484 .1562 .1641	.1979 .2083 .2187 .2292 .2396	.3750 .3958 .4167 .4375 .4583	.5625 ·5937	.6667 .7083 .750 .7917 .8333 .875 .9167 .9583	.9375	1.0625 1.125 1.1875 1.250 1.3125	1.3333 1.4167 1.500 1.5833 1.6667 1.750 1.8333 1.9167 2.000	2.0833 2.1875				

TAPERS PER FOOT IN INCHES AND CORRESPONDING ANGLES

===														
Taper per Foot	Included Angle			A: CE	NGLE W	ITH LINE	Taper per Foot					Angle with Center Line		
Taj Per	Deg.	Min.	Sec.	Deg.	Min.	Sec.	Taper per Foo	Deg.	Min.	Sec.	Deg.	Min.	Sec.	
$\begin{array}{c} 1 \\ 64 \\ \frac{1}{32} \\ 1 \\ \hline 16 \\ \frac{3}{32} \\ \hline \frac{1}{3} \\ 8 \\ \end{array}$	00000	4 8 17 26 35	28 58 54 52 48	0 0 0 0	2 4 8 13	14 29 57 26 54	I I 18 I 14 I 13 I 12	4 5 5 6 7	46 21 57 33 9	18 44 48 26	2 2 2 3 3	23 40 58 16 34	9 52 54 43 35	
$ \begin{array}{r} \frac{5}{3} \frac{2}{2} \\ \hline 16 \\ \hline 7 \\ \hline 32 \\ \hline 4 \\ \hline 9 \\ \hline 32 \\ \hline 4 \\ \hline 39 \\ \hline 3 \\ \hline 3 \\ \hline 2 \\ \hline 3 \\ 3 \\ \hline 3 \\ \hline 3 \\ \hline 3 \\ 3 \\ \hline 3 \\ \hline 3 \\ \hline 3 \\ 3 \\ \hline 3 $	0 0 0	44 53 2 11 20	44 44 34 36 30	0 0 0 0	22 26 31 35 40	22 52 17 48 15	1 5 8 1 4 7 8 2 2 4 4	7 8 8 9	44 20 56 31 42	48 26 2 36 42	3 4 4 4 5	52 10 28 45 21	24 13 1 48 21	
$\begin{array}{c} 5 \\ \overline{1} \\ 6 \\ 1 \\ \overline{3} \\ \overline{2} \\ 3 \\ \overline{3} \\ \overline{2} \\ 7 \\ \overline{1} \\ 6 \end{array}$	I I I 2	29 38 47 56 5	30 22 24 24 18	0 0	44 49 53 58 2	45 11 42 12 39	2 ¹ / ₂ 2 ³ / ₄ 3 ¹ / ₄ 3 ²	11 13 14 15 16	53 4 15 25 35	36 24 0 24 40	5 6 7 7 8	56 32 7 42 17	48 12 30 42 50	
$\begin{array}{c} \frac{1}{3} \frac{5}{2} \\ \frac{1}{2} \\ \frac{1}{2} \\ \frac{7}{3} \frac{2}{2} \\ \frac{1}{6} \\ \frac{1}{3} \frac{9}{2} \end{array}$	2 2 2 2 2	14 23 32 41 50	16 10 4 4 2	1 1 1	7 11 16 20 25	8 35 2 32 1	34 4 44 41 42 44	17 18 20 21 22	45 55 5 14 23	40 28 2 2 2	9 10 10	5 ² 27 2 37 11	50 44 31 1 41	
508 12216 32234	3 3 3· 3	59 7 16 25 34	42 56 54 50 44	I I I I	29 33 38 42 47	51 58 27 55 22	5 5 1 5 1 2 3 4 6	23 24 25 26 28	32 40 48 56 4	12 42 48 46 2	11 12 12 13 14	46 20 54 28 2	21 24 23 1	
5 23 67 27-j89 2 2 31 12 37-j89 2	3 4 4 4	43 52 1 10 19	44 38 36 32 34	I I 2 2 2	51 56 0 5	52 19 48 16 47	$\begin{array}{c} 6\frac{1}{4} \\ 6\frac{1}{2} \\ 6\frac{3}{4} \\ 7 \\ 7\frac{1}{4} \end{array}$	29 30 31 32 33	11 18 25 31 36	34 26 2 12 40	14 15 15 16 16	35 9 42 15 48	47 13 31 36 20	
15 16 31 32	4	28 37	24 20	2	14 18	12 40	7½ 7¾ 8	34 35 36	42 47 52	30 32 12	17 17 18	21 53 26	15 46 6	

Table for Computing Tapers

The Tabulated Quantities = Twice the Tangent of Half the Angle.

	,	,	, ,		.,		
Deg.	o'	10'	20'	30'	40 ′	50'	60′
0	.00000	.00290	.00582	.00872	.01164	.01454	.01746
1	.01746	.02036	.02326	.02618	.02910	.03200	.03492
2	.03492	.03782	.04072	.04364	.04656	.04946	.05238
3	.05238	.03528	.05820	.06110	.06402	.06692	.06984
4	.06984	.07276	.07566	.07858	.08150	.08440	.08732
5 6	.08732	.09024	.09316	.09606	.09898	.10190	.10482
	.10482	.10774	.11066	.11356	.11648	.11940	.12232
7 8	.12232	.12524	.12816	.13108 .14862	.13400	.13694	.13986
	.13986	.14278 .16034	.14570	.16618	.15156	.15448	.15740
9	.15740	.10034	.16326 .18084	.18378	.16912 .18670	.17204 .18964	.17498
10	.17498 .19258	.17790 .19552	.19846	.20138	.20432	.20726	.19258 .21020
12	.21020	.21314	.21610	.21904	.22198	.20/20	.22788
13	.22788	.23082	.23376	.23672	.23966	.24262	.24556
13	.24556	.24852	.25148	•25444	.25738	.26034	.26330
15	.26330	.26626	.26922	.27218	.27516	.27812	.28108
16	.28108	.28404	.28702	.28998	.29296	.29502	.29890
17	.29890	.30188	.30486	.30782	.31080	.31378	.31676
18	,31676	.31976	.32274	.32572	.32870	.33170	.33468
19	.33468	.33768	.34066	.34366	·34666	.34966	.35266
20	.35266	.35566	.35866	.36166	.36466	.36768	.37068
21	.37068	.37368	.37670	.37972	.38272	.38574	.38876
22	38876	.39178	.39480	.39782	.40084	.40388	.40690
23	.40690	.40994	.41296	.41600	.41904	.42208	.42512
24	.42512	.42816	.43120	•43424	.43728	.44034	.44338
25	•44338	.44644	.44950	.45256	.45562	.45868	.46174
26	.46174	.46480	.46786	.47094	.47400	.47708	.48016
27	.48016	.48324	.48632	.48940	.49248	.49556	. 49866
28	.49866	.50174	.50484	.50794	.51004	.51414	.51724
29	.51724	.52034	•52344	.52656	.52966	.53278	.53590
30	.53590	•53902	.54214	•54526	.54838	.55152	.55464
31	,55464	•55778	.56092	.56406	.56720	•57034	•57350
32	,57350	.57664	.57980	.58294	.58610	.58926	59242
33	-59242	.59560	.59876	.60194	.60510	.60828	.61146
34	61146	.61464	.61782	.62102	.62420	.62740	.63060
35	63060	.63380	.63700	.64020	.64342	.64662	.64984
36	64984	.65306	.65628	.65950	.66272	.66596	.66920
37	66920	.67242	.67566	.67890	.68216	.68540	.68866
38	68866	.69192	.69516	.69844	.70170	.70496	.70824
39	70824	.71152	.71480	.71808	.72136	.72464	.72794
40	72794	.73124	•73454	.73784	.74114	.74446	.74776
41	.74776	.75108	.75440	•75774	.76106	.76440	.76772
42	.76772	.77106	•77442	.77776	.78110	.78446	.78782 .80806
43	.78782 .80806	.79118	•79454	•79792	.80130	.80468	.82842
44	.82842	.81144	.81482	.81822	.82162	.82502	.84894
45	.02042	.83184	.83526	.83866	.84210	.84552	.04094

TABLE FOR COMPUTING TAPERS

The Tabulated Quantities = Twice the Tangent of Half the Angle.

	Labarate						
Deg.	o'	10'	20'	30 ′	40'	50'	60'
46	.84894	.85238	.85582	.85926	.86272	.86616	.86962
47	.86962	.87308	.87656	.88002	.88350	.88698	.89046
48	.89046	.89394	.89744	.90094	•90444	.90794	.91146
49	.91146	.91496	.91848	.92202	.92554	,92908	.93262
50	.93262	.93616	.93970	.94326	.94682	.95038	.95396
51	.95396	•95752	.96110	.96468	.96828	.97186	.97546
5^{2}	.97546	.97906	.9S268	.98630	.98990	•99354	.99716
53	.99716	1.00080	1.00444	1.00808	1.01174	1.01538	1.01906
54	1.01906	1.02272	1.02638	1.03006	1.03376	1.03744	1.04114
55	1.04114	1.04484	1.04854	1.05226	1.05596	1.05970	1.06342
56	1.06342	1.06716	1.07090	1.07464	1.07840	1.08214	1.08592
57	1.08592	1.08968	1.09346	1.09724	1.10102	1.10482	1.10862
58	1.10862	1.11242	1.11624	1.12006	1.12388	1.12770	1.13154
59	1.13154	1.13538	1.13924	1.14310	1.14696	1.15082	1.15470
60	1.15470	1.15858	1.16248	1.16636	1.17026	1.17418	1.17810
61	1.17810	1.18202	1.18594	1.18988	1.19382	1.19776	1.20172
62	1.20172	1.20568	1.20966	1.21362	1.21762	1.22160	1.22560
63	1.22560	1.22960	1.23362	1.23764	1.24166	1.24570	1.24974
64	1.24974	1.25378	1.25784	1.26190	1.26598	1.27006	1.27414
65	1.27414	1.27824	1.28234	1.28644	1.29056	1.29468	1.29882
66	1.29882	1.30296	1.30710	1.31126	1.31542	1.31960	1.32378
67	1.32378	1.32796	1.33216	1.33636	1.34056	1.34478	1.34902
68.	1.34902	1.35326	1.35750	1.36176	1.36602	1.37028	1.37456
69	1.37456	1.37984	1.38314	1.38744	1.39176	1.39608	1.40042
70	1.40042	1.40476	1.40910	1.41346	1.41782	1.42220	1.42658
71	1.42658	1.43098	1.43538	1.43980	I.44422	1.44864	1.45308
72	1.45308	1.45754	1.46200	1.46646	1.47094	1.47542	1.47992
73	1.47992	1.48442	1.48894	1.49348	1.49800	1.50256	1.50710
74	1.50710	1.51168	1.51624	1.52084	1.52544	1.53004	1.53466
75	1.53466	1.53928	1.54392	1.54856	1.55322	1.55790	1.56258
76	1.56258	1.56726	1.57196	1.57668	1.58140	1.58612	1.59088
77	1.59088	1.59562	1.60040	1.60516	1.60996	1.61476	1.61966
78	1.61956	1.62440	1.62922	1.63406	1.63892	1.64380	1.64868
79	1.64868	1.65356	1.65846	1.66338	1.66830	1.67324	1.67820
80	1.67820	1.68316	1.68814	1.69312	1.69812	1.70314	1.70816
81	1.70816	1.71320	1.71824	1.72332	1.72836	1.73348	1.73858
82	1.73858	1.74368	1.74882	1.75396 1.78506	1.75910	1.76428	1.76946
83	1.76946	1.77464	1.77984	1.78506	1.79030	1.79554	1.80080
84	1.80080	1.80608	1.81138	1.81668	1.82198	1.82732	1.83266
85	1.83266	1.83802	1.84340	1.84878	1.85418	1.85960	1.86504
86	1.86504	1.87048	1.87594	1.88142	1.88690	1.89240	1.89792
87	1.89792	1.90346	1.90902	1.91458	1.92016	1.92576	1.93138
88	1.93138	1.93700	1.94266	1.94832	1.95400	1.95968	1.96540
80	1.96540	1.97112	1.97686	1.98262	1.98840	1.99420	2.00000
90	2.			1	1		

Refer to page 364 for explanation of table.

TABLE FOR USE IN COMPUTING TAPERS

In the table on pages 362 and 363 the quantities when expressed in inches represent the taper per inch corresponding to various angles advancing by 10 minutes from 10 minutes to 90 degrees. If an angle is given as, say, 27½ degrees and it is desired to find the corresponding taper in inches, the amount, 0.4894 may be taken directly from the table. This is the taper per inch of length measured as in Fig. 6, along the axis. The taper in inches per foot of length is found by multiplying

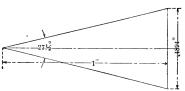


Fig. 6. — Taper per Inch and Corresponding Angle

the tabulated quantity by 12, and in this particular case would be 0.4894" \times 12 = 5.8728". Where the included angle is not found directly in the table, the taper per inch is found as follows: Assume that the angle in question is 12½ degrees, then the nearest angles in the table are 12° 10′ and 12° 20′, the respective quantities tabulated under these angles being 0.21314 and 0.21610. The difference between the two is 0.00296, and as 12½° is half way between 12° 10′ and 12° 20′ one half of 0.00296, or 0.00148 is added to 0.21314, giving 0.21462" as the taper of a piece 1 inch in length and of an included angle of 12½ degree. The taper per foot equals 0.21362" \times 12 = 2.5634".

TABLE FOR DIMENSIONING DOVETAIL SLIDES AND GIBS

THE table on page 365 is figured for machine-tool work, so as to enable one to tell at a glance the amount to be added or subtracted in dimensioning dovetail slides and their gibs, for the usual angles up to 60 degrees. The column for 45-degree dovetails is omitted, as A

and B would, of course, be alike for this angle.

In the application of the table, assuming a base with even dimensions, as in the sketch Fig. 7, to obtain the dimensions x and y of the slide Fig. 8, allowing for the gib which may be assumed to be $\frac{1}{2}$ inch thick, the perpendicular depth of the dovetail being $\frac{5}{8}$ inch, and the angle 60 degrees, look under column A for $\frac{5}{8}$ inch and it will be found opposite this that B is 0.360 inch, which subtracted from 2 inches gives 1.640 inches, the dimension x. To find y first get the dimension 1.640 inches, then under the column for 60-degree gibs (where C is $\frac{1}{8}$ inch), D is found to be 0.289 inch, which is added to 1.640, giving 1.929 inches.

In practice this dimension is usually made a little larger, say to

the nearest 64th, to allow for fitting the gib.

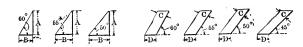


TABLE FOR DIMENSIONING DOVETAIL SLIDES AND GIBS

A	В	В	В	С	D	D	D	D
1 " 3 2	.018″	.022"	.027"		.144"	.152"	.163"	.176″
$\frac{1}{16}''$.036″	.044″	.053"	3 m	.216″	.228″	.244"	.264″
1 "	.072"	.087"	.105″	1/4"	.289″	.305″	.326″	·353"
<u>1</u> "	.144"	.175"	.210"	5 " 16"	.361"	.381″	.407"	.442"
3"	.216"	.262″	.314"	3"	•433"	·457"	.489″	.530"
1/2"	.288″	.350″	.420″	1/1	•577"	.610″	.652"	.707"
<u>5</u> "	.360″	·437"	.525"	• <u>5</u> "	.721"	.762"	.815"	.883"
<u>3</u> "	•433"	.525"	.629"	3″ 4	.866″	.915"	•979″	1.060"
<u>7</u> "	.505"	.612"	•734"	7/1	1.010"	1.067"	1.142"	1.237"
1"	•577"	.700″	.839"	ı"	1.154"	1.220"	1.305"	1.414"
1 ½"	.649"	.787″	.944″			!		
1 1/4"	.721"	.875"	1.049"		23."	1 -	1 03/	~ ,
1 3″	•794″	.962"	1.153"		2"		2"	
$_{1\frac{1}{2}''}$.866″	1.050"	1.259"	·Ţ			W	7
$1\frac{3}{4}''$	1.010"	1.225"	1.469"	3%	ŝ			1
2"	1.154"	1.400″	1.677"					
21/1	1.298"	1.575"	1.888"			FIG		
21"	1.442"	1.750"	2.097"	ammun		FIG. 7		vuuuu
2 ³ / ₄ "	1.588"	1.925"	2.307"					
3"	1.732"	2.100"	2.517"	100	100		127	
32"	2.020"	2.450"	2.937"		x		<u>, </u>	
4"	2.308″	2.800"	3.356"		1-6	40"1-	929"	į
42"	2.598"	3.150"	3.776"		-2%/"	+	23/4-	
5 "	2.885"	3.501"	4.195"	<u>. </u>		F1 Ġ. 8		

MEASURING EXTERNAL AND INTERNAL DOVETAILS

THE accompanying table of constants is for use with the plug method of sizing dovetail gages, etc. The constants are calculated for the plugs and angles most in use; and to use them a knowledge of arithmetic is all that is required. The formulas by which they were obtained are added for the convenience of those who may have an unusual angle to make.

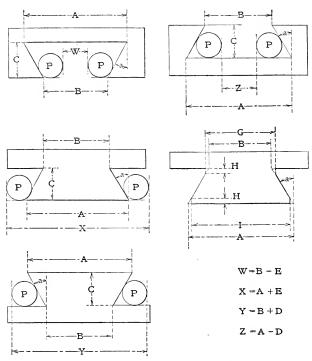


Fig. 9. — External and Internal Dovetails

As an example of the use of the table, suppose that Z, Fig. 9, is the dimension wanted, and that the dimension A and the angle a are known. A glance at the formulas above shows that Z=A-D. Then the constant D corresponding to the size of plug and the angle used is subtracted from A and the remainder equals Z. For instance, if A=4'', the plug used $=\frac{3}{8}''$, and the angle =30 degrees, then Z=A-D=4''-1.0245''=2.9755''.

If A is not known but B and C are given, according to the formula below the table A=B+CF. Then if B=3.134'', $C=\frac{3}{4}''$, and the angle is 30 degrees, as before, $A=B+CF=3.134''+(.75''\times 1.1547)=4''$, whence Z can be found, as already shown.

If the corners of the dovetail are flat, as shown in Fig. 9 at I and G, and the dimensions I and H and the angles are known, it will be found from the formulas below the table that A also = I + HF; so that if I = 3.8557'', $H = \frac{1}{8}''$, and the angle = 30 degrees, then $A = I + HF = 3.8557'' + (.125'' \times 1.1547) = 4''$, from which Z is found as before.

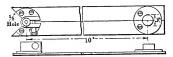
Constants for Dovetalis

Pl	ıg	60°	55°	5°°	45°	40°	35°	300
1 ″	D E	1.1830 .3170	1.0429 •3 ² 33	.9368 .3410	.8535 •3536	.7861 .3666	-7302 -3802	.6830 •3943
3″ 8	D E	1.7745 .4755	1.5643 •4932	1.4053 .5115	1.2803 •5303	1.1792 •5499	1.0954 •5702	1.0245 •5915
$\frac{1}{2}''$	D E	2.3660 .6340		1.8730 .6820	1.7070 .7072	1.5722 •7332	1.4604 .7603	1.3660 .7886
<u>3</u> "	D E	3.5490 .9510	3.1286 .9864		2.5606 1.0606	2.35 84 1.0998	2.1903 1.1404	2.0490 1.1830
	F	3.4641	2.8563	2.3836	2	1.6782	1.4004	1.1547

A = B + CF = I + HF
B = A - CF = G - HF
E = P
$$\left(\cot \frac{90 + a}{2}\right)$$
 + P
D = P $\left(\cot \frac{90 - a}{2}\right)$ + P
F = 2 tan a

TOOL FOR LAYING OUT ANGLES ACCURATELY

The bevel gage here shown is for laying out angles accurately. In using this gage set a vernier caliper or large "micrometer" to twice the sine of half the angle desired, multiplied by ten, add one



half inch and open the gage till it fits the vernier; this gives the angle within the limits of the measuring tool and the radius of the gage. The eighth-inch hole in the center is for a setting plug when it is desirable to lay out an angle from a given center.

The table gives the measurements over the half disks required for setting the arms of the gage to give any angle from 1 to 45 degrees, and also the setting for any number of holes in a circle from 3 to 22.

TABLE FOR SETTING TOOL FOR LAYING OUT ANGLES

		GAGE	SETTING FO	r Even	Degrees		
Angle Degrees	Measure- ment Over Disks	Angle De- grees	Measure- ment Over Disks	Angle De- grees	Measure- ment Over Disks	Angle De- grees	Measure- ment Over Disks
1 2 3 4 5 6 7 8 9 10	0.6746 0.8490 1.0236 1.1980 1.3724 1.5468 1.7210 1.8952 2.0692 2.2432 2.4170	12 13 14 15 16 17 18 19 20 21 22	2.5906 2.764 2.9374 3.1106 3.2834 3.4562 3.6286 3.8010 3.9730 4.1448 4.3162	23 24 25 26 27 28 29 30 31 32 33	4.4874 4.6582 4.8288 4.9080 5.1690 5.3384 5.5176 5.6764 5.8448 6.0128 6.1804	34 35 36 37 38 39 40 41 42 43 44 45	6.3474 6.5142 6.6804 6.846 7.0114 7.1762 7.3404 7.5042 7.6674 7.830 7.9922 8.1536

GAGE SETTINGS FOR HOLES IN A CIRCLE

No. of Holes in Circle	Measure- ment Over Disks	No. of Holes in Circle	Measure- ment Over Disks	No. of Holes in Circle	Measure- ment Over Disks	No. of Holes in Circle	Measure- ment Over Disks
3	17.8206	8	8.1536	13	5.2864	18	3.9730
4	14.6422	9	7.3404	14	4.9504	19	3.7918
5	12.2558	10	6.6802	15	4.6582	20	3.6286
6	10.5	11	6.1346	16	4.4018	21	3.4808
7	9.1776	12	5.6762	17	4.1750	22	3.3462

SHOP AND DRAWING ROOM STANDARDS

STANDARD JIG PARTS

Drill Bushings

When drilling and reaming operations are to be performed in the same jig, two slip bushings, one for the drill and the other for the reamer, should be used; if the jig is to be used for a large number of parts, the hole for the bushings should in turn be bushed with a steel lining to prevent wearing. The soft cast-iron will wear rapidly if this is not done, and the jig will soon have to be rebored and rebushed.

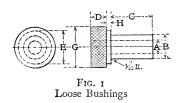
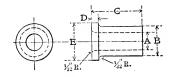


Fig. 2 Fixed Bushings

		Loose	Bushi	NGS			Fixer	Bush	INGS
A	В	С	D	Е	G	Н	A	В	С
No. 30 2 14 5 15 3 4 5 5 6 5 6 5 6 5 6 5 6 5 6 6 6 6 6 6 6	14.50 GRO 16.00 GRO 15.00	9.76.58.636.11.6.34.634.8.16.78.78.5.66 I I I I I I I I I I I I I I I I I I I	14-14-5 0 5 5 1 3 20 20 20 27 1 7 1 12 1 2 9 1 9 1 20 20 1 1 0	7.6 2.6 1.1 2.6 1.1 1.1 1.1 1.1 1.1 1.1 1.1 1.1 1.1 1	9 6 8 14 6 8 6 7 7 8 5 6 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	1 16 16 16 16 16 16 16 16 16 16 16 16 16	52 30 12 14563857615881166 243361778 1156 1	1445 G 350 7 G 1220 G 558 T G 344 7 G 5 G T T T G 8 G G G G G G G G G G G G G G	0 5 6 10 10 10 10 10 10 10 10 10 10 10 10 10

370

Three different styles of bushings with their dimensions are shown in Figs. 1, 2 and 3. These can be blanked out in quantities and finished to required sizes as needed, and should be made of tool Allowances should be made in the blanks for grinding and



A	В	С	D	E	A	В	С	D	E
145 6 380 6 10 29 6 58	1,6 1,20,10 5/8 1,6 3/4 7/8	116 34 35636 78 156 116 314 35636 78 156 116 1	\$\[\text{12} \\ \	9 1558 116 34 116 178 178 116 176	116 34 116 78 116 116 1	15 I I 176 I 178 I 178 I 178 I 178 I 178 I 178	$ \begin{array}{c} 1\frac{1}{16} \\ 1\frac{8}{8} \\ 1\frac{3}{16} \\ 1\frac{4}{16} \\ 1\frac{5}{16} \\ 1\frac{5}{16} \end{array} $	1801801805 355 35 35 35 35 35 35 35 35 35 35 35 3	1 \frac{1}{8} \\ 1 \frac{3}{1 \cdot 6} \\ 1 \frac{4}{1 \cdot 6} \\ 1 \frac{45}{1 \cdot 6} \\ 1 \frac{3}{1 \cdot 6} \\ 1 \frac{7}{1 \cdot 6} \\ 1 \

Fig. 3. - Fixed Bushings

lapping after hardening. Fig. 1 shows a slip bushing; Fig. 2 a stationary bushing, and Fig. 3 a stationary bushing where tools with stop collars are to be used. Such bushings as shown in Figs. 2 and 3 are also used for linings for slip bushings.

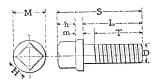


Fig. 4 Collar-Head Jig Screws.

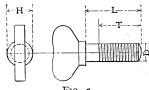


Fig. 5 Winged Jig Screws

D	Thrd.	L	Т	h	m	н	М	S	D	Thrd.	Н	L	Т
$\begin{array}{c} \frac{1}{4} \\ \frac{5}{16} \\ \frac{3}{8} \\ \frac{7}{16} \\ \frac{1}{2} \end{array}$	20 18 16 14	I I I 1 ½ 2 2	34 34 I I 131 I 12	$\frac{1}{4}$ $\frac{1}{5}$ $\frac{1}{6}$ $\frac{3}{1}$ $\frac{5}{1}$ $\frac{1}{6}$	8 3 1 8 1 8 1 6 1 6	145 6 13807 6 112	7 16 12 5 8 116 16 16 16 16 16 16	$ \begin{array}{c} 1\frac{1}{3}\frac{1}{2} \\ 1\frac{7}{16} \\ 2 \\ 2\frac{5}{8} \\ 2\frac{1}{16} \end{array} $	1 4 5 16 3 8	20 18 16	38 7 16 12	I I I ¹ / ₂	34 34 I

Binding Screws

Binding-screws should be made in various sizes and with threads to conform to the standard taps with which the shop is provided. When drills of a very large size are used, a screw with a square or hexagon head is best, as the work requires firm clamping. If the drills used are small, a winged screw will be sufficient and more convenient, as it will require less time to manipulate. Some good screws for clamping straps are shown in Figs. 4 and 5. Of course the screws can be made of any length desired.

When the work is to be held against the seat or a stop by means of a set-screw, such screws as shown in Figs. 6 and 7 will be found very useful. If, however, the work is very light, a wing screw can

be used.

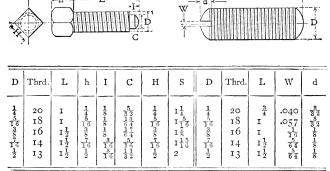


Fig. 6 Square-Head Jig Screws

Fig. 7 Headless Jig Screws

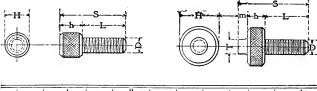
Supporting Screws

Figs. 8 and 9 show screws that are useful in supporting work against the thrust of drills when the work is of such a nature that it cannot be supported otherwise.

Locking Screws

A convenient hinge-cover locking screw is shown in Fig. 10. This screw, when used, should be adjusted so that only a quarter turn will be needed to clamp or release the cover, which should be slotted to admit the head of the screw.

The different sizes of the styles of screws shown are not only used with drilling jigs, but are equally useful with other jigs and fixtures. These screws should be made of screw stock and case-hardened.



D	Thrd.	L	h 	s	н	D	Thrd.	L	h	m	s	н	Т
16 16 3	20 18 16	3 1 1	38 1 2 9 16	1 ½ 1 ½ 1 ½ 1 ½	1 2 9 16 5 8	1 4 5 16 3 8	20 18 16	34 I I	$\frac{5}{16}$ $\frac{3}{8}$ $\frac{7}{16}$	$\frac{\frac{3}{16}}{\frac{7}{32}}$	1 1 1 9 1 1 1 1 6	34 78 I	14 5 16 38

Fig. 8 Nurled-Head Jig Screws

Fig. 9 Nurled-Head Jig Screws

Strap Dimensions

A convenient strap to use with these jigs is shown in Fig. 11. The straps should be made of bessemer steel and case-hardened after finishing. The slot G can be located in the proper position and made of such dimensions as to allow the strap to be slipped back out of the way when work is being placed in and taken from the jig.

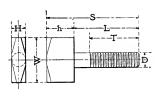


Fig. 10. - Locking Jig Screws

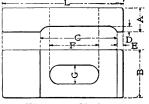
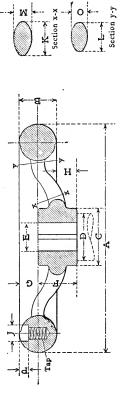


Fig. 11. — Jig Straps

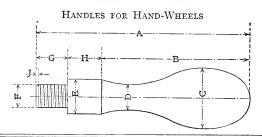
D	Thrd.	Н	h	L	S	Т	w	A	В	С	D	Е	L
16 3 8 7 16 2	18 16 14 13	16 38 7 16 12	508 116 004 0014	I 1/25/207/207/207/207/207/207/207/207/207/207	2 100 5 60 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	$ \begin{array}{c} 1\frac{1}{16} \\ 1\frac{3}{16} \\ 1\frac{5}{16} \\ 1\frac{5}{16} \end{array} $	1212250000000	I I I 14 I 14 I	158 158 2 2	1010010010010	1 8 1 8 1 1 6 3 1 6 3 1 6 3 1 6 3 1 6 3 1 6	2½ 3 3½ 3½ 3½ 4



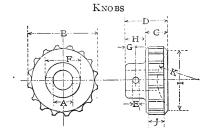
HAND WHEELS. (Pratt & Whitney Co.)

,		37
	No. of Arms	~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~~
	Handle No.	00 H H U W 4 4 N N N
	Spline	
	Tap	### ### ##############################
	Ъ	ap-14-14-14-pp-pp-pp-pp-pp-pp-pp-pp-pp-pp-pp-pp-pp
	0	who had the thought with the the
	×	HALISHEN WALLE HOSTO TO TO THE TOTAL TO THE
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374 SHOP AND DRAWING ROOM STANDARDS

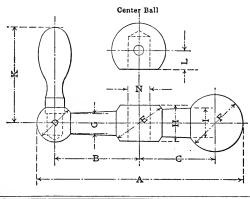


No.	A	В	С	D	Е	F	G	Н	J	P. I.
00 0 1 2 3 4 5 6	$\begin{array}{c} 1\frac{1}{1}\frac{5}{1}\frac{1}{1}\frac{1}{1}\frac{1}{1}\frac{5}{1}\frac{5}{1}\frac{1}{1}\frac{5}{1}\frac{5}{1}\frac{1}{1}\frac{5}{1}\frac{5}{1}\frac{5}{1}\frac{5}{1}\frac{1}{1}\frac{5}{1}\frac$	$\begin{array}{c} 1 \frac{5}{11556} \\ 1 \frac{1}{1156} \\ 2 \\ 2 \frac{38}{116} \\ 3 \frac{3}{116} \\ 3 \frac{5}{116} \\ 3 $	17/25/65/65/6 11/25/65/6 1 18/14/7/6 1 17/6	14 1(21 3)(2 7 16 5)(2 1/2 0) (6 5)(8 1/2 1/2 0) (7 5)(8 1/2 1/2 0) (16 5)(8 1	3/8 15/315/31 1/2 9/24.5/31 1/82 3/4	5.66 5.66 5.88 5.88 5.88 5.88 5.88 5.88	3/8 3/8 3/8 3/8 1/2 1/2 1/2 1/2 1/6	14 3 8 9 6 9 6 5 8 5 8 5 8 5 8 5 8 5 8 5 8 5 8 5 8 5	1 32 32 1 32 1 32 1 32 1 32 1 32 1 32 1	28 28 24 24 24 24 24 20 16



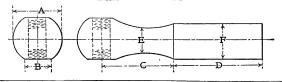
A	В	С	D	E	F	G	н	I	J	K
8. 6 145 6 38 7 6 12 9 7 8 8 16 24	I	9 6 5 6 6 7 6 7 6 7 6 9 6 9 6 7 6 7 6 7 6 1 6 1 6 1 6 1 6 1 6 1 6 1	1521(616778787816 111(6141414 111(41414	3 51 52 52 53 53 53 53 53 53	1,60 16 3,61 16 1,61 16 7,87,8 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	18 3 6 3 1 7 2 7 5 1 4 1 4 5 2 9 2 12 12 12 12 12 12 12 12 12 12 12 12 1	14 38 38 87 6 17 16 12 12 9 6 9 6 9 6 9 6 9 6	$\begin{array}{c} \frac{15}{16} \\ 1\frac{1}{16} \\ 1\frac{1}{16} \\ 1\frac{1}{36} \\ 1\frac{1}{36} \\ 1\frac{1}{36} \\ 1\frac{1}{36} \\ 1\frac{1}{36} \\ 2\frac{1}{16} \\ 2\frac{1}{16} \\ 2\frac{1}{16} \\ \end{array}$	5.03 THE FLE PRO PRIOR 7 TO THE PROPRIOR DISC	Cup Out to Suit

BALL HANDLES
(Pratt & Whitney Co.)



A	В	С	D	Е	F	G	н	I	к	L	N
3 4 4½ 5 5 5 6 7 8	$\begin{array}{c} 1 \frac{3}{16} \\ 1 \frac{3}{16} \\ 1 \frac{3}{2} \\ 2 \frac{1}{3} \\ 2 \\ 2 \\ 2 \\ 2 \\ 2 \\ 2 \\ 3 \\ 3 \\ 7 \\ 6 \\ \end{array}$	$\begin{array}{c} I \\ I \frac{7}{16} \\ I \frac{5}{18} \frac{3}{3} \\ I \frac{1}{3} \frac{1}{3} \frac{7}{7} \\ 2 \frac{1}{3} \frac{1}{7} \frac{1}{2} \\ 2 \frac{2}{3} \frac{2}{3} \\ 3 \\ 3 \\ 3 \\ \end{array}$	558 146 34478 15/6 1 1 1 1 1 1 1 1 1 1 1 8	$\begin{array}{c} \frac{7}{8} \\ I \\ I \frac{3}{16} \\ I \frac{3}{16} \\ I \frac{1}{16} \\ I \frac{1}{16} \\ I \frac{7}{16} \\ I \frac{1}{2} \end{array}$	I I 18 I 14 I 18 I 176 I 176 I 1 166 I 1 166 I 1 166 I 1 166 I 1 166 I I 1 166 I I 1 166 I I I 1 166 I I I I I I I I I I I I I I I I I I I	3 83 67 61 121 229 69 6	9 16 116 34 136 78 156 16 1 116 1	12965 58116 116 116 116 78	1 78 1 1 1 56 2 1 1 1 6 2 1 1 1 6 2 1 1 1 6 2 1 1 1 6 3 1 1 1 6 3 1 1 1 6 3 1 1 1 6 3 1 1 1 6	9.21.12 3 8 3 2 7 6 5 2 2 7 2 1 3 1 2 7 2 2 3 2 3 2 3 2 3 2 3 2 3 2 3 3 2 3	3/87/6 2 12 9/6 5/8 5/8 5/8

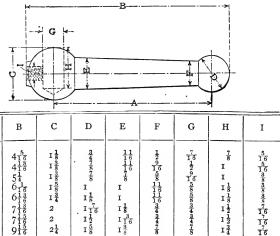
BINDER HANDLES

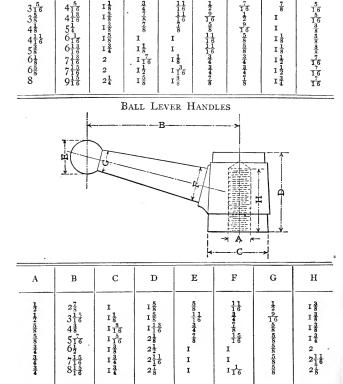


A	В	С	D	E	F	Dia. of Tap
I I 1 1 6 I 1 2 I 1 2	3 18 18 18 18	$ \begin{array}{c c} \hline 1\frac{5}{8} \\ & 1\frac{3}{4} \\ 2\frac{1}{8} \\ & 2\frac{5}{8} \end{array} $	$ \begin{array}{cccccccccccccccccccccccccccccccccccc$	7 16 9 16 5	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	$\frac{7}{16}$ $\frac{9}{16}$ $\frac{1}{2}$ $\frac{9}{8}$ $\frac{1}{16}$

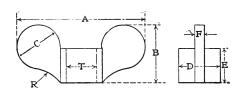
A





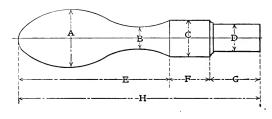


WING NUTS



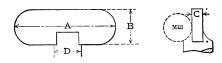
A	В	С	D	E	F	R	Т
$\begin{array}{c} \frac{192}{1523} \\ \frac{1}{15} \\ \frac{1}{16} \\ 1\frac{2}{1} \\ 1\frac{5}{16} \\ 1\frac{1}{16} \\ 2\frac{1}{16} \end{array}$	11077-1077-1077-1077-1077-1077-1077-107	7 32 16 38 12 9 16 5 16 5 16 38 16 38 16 38 16 38 16 38 16 38 38 16 38 38 38 38 38 38 38 38 38 38 38 38 38	7 14 14 12 12 58	$\begin{array}{c} \frac{5}{32} \\ \frac{1}{4} \\ \frac{3}{8} \\ \frac{7}{16} \\ \frac{1}{2} \\ \frac{1}{16} \\ \frac{1}{16} \end{array}$	$\frac{3}{3}$ $\frac{1}{2}$ $\frac{1}{8}$ $\frac{1}{8}$ $\frac{5}{3}$ $\frac{5}{2}$ $\frac{5}{3}$ $\frac{5}$	$\begin{array}{c} \frac{9}{3^{2}} \\ \frac{3}{3^{2}} \\ \frac{5}{16} \\ \frac{7}{16} \\ \frac{7}{16} \\ \frac{1}{2} \\ \frac{3}{4} \end{array}$	18 5 32 4 5 16 3 7 16 12

MACHINE HANDLES



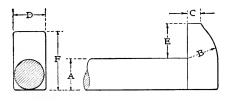
A	В	С	D	Е	F	G	н
$\begin{array}{cccccccccccccccccccccccccccccccccccc$	$\begin{array}{c c} \frac{1}{3}\frac{1}{2} & & \\ \frac{1}{3}\frac{2}{2} & & \\ \frac{7}{16} & & \\ \frac{1}{3}\frac{5}{2} & \\ \frac{1}{2} & & \\ \frac{9}{16} & \\ \frac{5}{8} & & \\ \end{array}$	152 152 153 153 153 153 153 153 153 153 153 153	5 16 5 16 5 16 5 16 5 7	$ \begin{array}{r} 1\frac{15}{16} \\ 2\\ 2\frac{3}{8} \\ 2\frac{13}{16} \\ 3\frac{1}{8} \\ 3\frac{7}{16} \\ 3\frac{5}{8} \end{array} $	7 16 58 58 11 16 34 78 155 16	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	2 ⁷ / ₈ 3 ¹⁵ / ₁₆ 3 ²⁴ / ₁₁ / ₆ 4 ¹¹ / ₁₈ 5 ¹ / ₁₆

THUMB NUTS



D	A	В	С	Mill
16 14 14 15 16 15 16 15 16 15 16 15 16 15 16 15 16 15 16 16 16 16 16 16 16 16 16 16 16 16 16	$\begin{array}{cccccccccccccccccccccccccccccccccccc$	14 5 16 2 3 7 16 12 2 9 16 5 5 11 1 6	$\begin{array}{cccccccccccccccccccccccccccccccccccc$	5 16 7 16 7 16 1 2 1 2 1 2 5 16 17 17 17 17 12 5 5 18 18 18 18 18 18 18 18 18 18 18 18 18

HOOK BOLTS



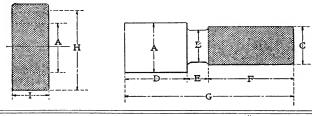
Dia. of Bolt	Thickness of Head	Thickness at End	Width of Head	Off Set of Head	Length of Head
A	В	С	D	E .	F
35012350374518 I	766 9166 1166 1155 1165 116	3 6 14 5 6 c. 887 6 7 1 1 2	3)0 1/215/00 3/47 100 I	7.60 61.000056 11.000056 11.600056	$\begin{array}{c} \frac{13}{16} \\ 1\frac{1}{16} \\ 1\frac{5}{16} \\ 1\frac{5}{16} \\ 1\frac{13}{16} \\ 2\frac{1}{16} \end{array}$

COUNTERBORES WITH INSERTED PILOTS

} 		Diam.	K	L	М	N	0
	M. W. M.	3 7 16 1 2 9 16 5	1 3 2 7 6 6 1 3 2 7 1 3 2 7 1 3 2 7 1 3 2 7 1 3 2 7 1 3 2 7 1 3 2 7 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	1 4 3 8	$1\frac{15}{32}$	3 16 5 16	From $\frac{3}{16}''$ to $\frac{15}{32}''$ in 32nds.
		38776 11229 1558 1134 350 78 566	" " "	" " " " " " " " " " " " " " " " " " "	 	46 46 46	From $\frac{1}{2}''$ to $\frac{3}{3}\frac{1}{2}''$ in 32nds.
% to I'Counterbores to	have 4 Flutes	I I 1 1 6 I 8 I 1 8 I 1 4 I 1 4 I 1 5 I 1 6 I 1 7 I 1	" 2\frac{1}{3}\frac{1}{2} " " " " " " " "	 1 2 	2 ² / ₃ ⁷ / ₂	1 2 	From 1" to 1 ¹ / ₃ 5" in 32nds.

Diam. A	В	С	D	Е	F	G	н	I	J
327 - 20 - 0 - 0 - 0 - 0 - 0 - 0 - 0 - 0 -	3 16 16 16 16 16 16 16 16 16 16 16 16 16	Total plant is the control of the co	56 16 1	$\begin{array}{c} \mathbf{I} \\ $	$\begin{array}{cccccccccccccccccccccccccccccccccccc$	3 3 3 4 4 4 4 4 5 5 5 5 5 6 6 6 6 6 7 7	3 16 16 16 16 16 16 16 16 16 16 16 16 16	1 1 5 7 1 1 5 7 1 1 1 5 7 1 1 1 1 1 1 1	114 14 178 178 118 119 119 119 119 119 119 11

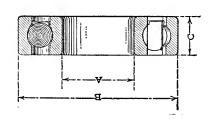
DIMENSIONS OF STANDARD PLUG AND RING GAGES



								i
			Plug				Rı	NG
Dia. of Plug	Dia. of Neck	Dia. of Handle	Length of Std. Plug	Length of Neck	Length of Handle	Total Length of Gage	O. S. Dia. of Ring	Thickness of Ring
A	В	С	D	Е	F	G	Н	I
145[2397]6-29[355]1624-26755[6] I I I I I I I I 2 2 2 2 3	7 10 10 16 16 16 16 16 16 16 16 16 16 16 16 16	7-17-17-17-19-17-19-18-18-18-18-18-18-18-18-18-18-18-18-18-	$\begin{array}{cccccccccccccccccccccccccccccccccccc$	$\frac{5}{10}, \frac{1}{10}, \frac{1}$	-10-14-00-01-00-04-00 -10-10-14-14-14-00-00-00-00-00-00-00-00-00-00-00-00-00	3 3 3 3 4 4 4 4 5 5 5 5 5 5 5 5 5 5 5 5	I I I I I I I I I I I I I I I I I I I	10.0010, 10.000 min 10

"HB"—"DWF" RADIAL BEARINGS—MEDIUM 300 SERIES (HESS-BRIGHT MFG. CO.)

	Ball	Dia.	-4-	4	To a	7 P	mino	I'd	400	oli,	oleo j	19	m -e	miso,	r-ja0	S (C)	н	11/4	₩	II's	₹ H	1,6	e H	₹ I
	linute	1500	125	130	220	240	350	485	615	770	970	1100	1320	1540	1760	1980	.2245	2530	2750	3025	3300	3520	3850	4620
	v. per M	1200	140	155	240	285	395	530	999	835	1045	1210	1485	1705	1980	2255	2530	2860	3190	3520	3850	4180	4510	5390
	and Re	800	160	175	285	330	485	625	770	066	1265	1485	1760	2000	2420	2750	3080	3520	3850	4290	4730	5170	5610	6710
	Load in Lbs. and Rev. per Minute	200	185	210	330	375	550	725	925	1165	1430	1715	2070	2420	2795	3190	3630	4070	4510	5015	2610	6050	0099	7875
vo	Load	200	220	265 285	395	440	999	880	0011	1430	1760	2000	2530	2970	3410	3895	4400	4995	5500	919	6820	7435	8140	0896
Medium Series	Width Ċ	inches	0.4331	0.4724	0.5512	9065.0	0.0003	0.7480	0.8268	0.0025	0.0843	1.0630	1.1417	1.2205	1.2992	1.3780	1.4567	I.5354	1.6142	1.6929	1.7717	1.8504	1.9291	1.9685
Medi	Wic	mm.	11	12	14	15	17	61	21	23	25	27	50	31	33	35	37	39	41	43	45	47	46	20
	Diameter B	inches	1.3780	1.4567 1.6536	1.8504	2.0473	2.4410	2.8347	3.1490	3.5433	3.9370	4.3307	4.7244	5.1181	5.5118	5.9055	6.2992	6.6929	2.0866	7.4804	7.8741	8.4646	8.8583	9.4489
	Dian	mm.	35	37	47	52	02	72	8	8	8	OII	120	130	140	150	991	170	180	061	200	215	225	240
	Bore A	inches	0.3937	0.4724	0.6693	0.7874	0.0843	1.181.1	1.3780	1.5748	1.7717	1.9085	2.1654	2.3622	2.5591	2.7559	2.9528	3.1496	3.3465	3.5433	3.7402	3.9370	4.1339	4.3307
		mm.	01	12	17	20	25	30	35	40	45	20	55	8.	65	20	7.5	8	85	8	95	8	105	011
	No. of	Bearing	300	301 302	303	304	305	300	307	308	300	310	311	312	313	314	315	310	317	318	319	320	321	322
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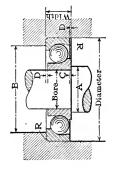
"HB"-"DWF" THRUST COLLAR BEARINGS -- MEDIUM WEIGHT SERIES NO. 1100

(HESS-BRIGHT MFG. CO.)

"HB"-"DWF" THRUST COLLAR BEARINGS — MEDIUM WEIGHT SERIES NO. 1100 — Continued

(HESS-BRIGHT MFG. CO.)

																									J
	Badhs	Dia.	-	∀ -√<	-14	-4*	~	η G	الم	13/30	69(3	r'2	1,6	~≈	40	-(0	ار اها	ed at	NO/08	~=	-(1	n w	50	1-100	I
with Ball Cage	ag a	No.	00	01	13	91	18	17	18	17	IO	18	10	18	10	20	10	61	20	01	20	61	10	61	19
un 18a		i.	0.50	0.53	0.65	0.57	0.70	0.0.1	0.88	1.12	1.30	1.33	1.50	1.52	1.69	1.85	I.0.I	2.16	2.33	2.37	2.76	2.72	2.04	3.16	3.64
5	×	mm.	0	13.4	16.6	I4.4	17.8	23.8	22.4	28.5	33	33.8	38.1	38.5	42.8	47.I	48.5	54.8	50.2	00.3	70.2	69.2	74.8	80.2	92.5
		'n.	9	o	0.	i,	0.	90.	90.	0.	90.	90.	90.	80.	80.	80.	8c.	80.	oI.	01.	10	oI.	.12	.12	.12
	1	mm.	н	н	н	н	ı	1.5	1 5	1.5	1.5	1.5	1.5	62	¢1	2	2	63	2.5	2.3	2.5	2.5	3	3	٤
		in.	0.08	1.18	1.38	1.38	1.57	1.07	1.97	2.30	2.56	2.76	2.05	3.15	3.35	3.54	3.74	4.13	4.33	4.53	4.92	5.12	5.51	16.5	69.9
	R	mm.	25	30	33	35	04	20	20	9	65	70	7.5	000	85	96	95	105	OII	115	125	130	140	150	170
	U	i.	10 X	L 3	21,6	2	2 8 8	2 5	316	3 2	5.00 50.00	4 *	-(+. -(+.	4	413	415	n.e	516	9	e.×		7	7 21	20	6
	E	ii.	e,	-124	- 9	r-iao	I 8	H	- I	116	1 3	21,6	2	21,0	23 70,00	213	3	313	C3 8/8	ς.) π)π	318	4	4 e×	4.3	ν ω æ
		.ë	0.47	0.07	0.87	90'I	1.26	1.46	1.05	1.85	2.05	2.24	2.44	2.04	2.83	3.03	3.23	3.46	3.66	3.86	4.06	4.25	4.65	5.04	5.63
ĺ	E	mm.	12	17	2.2	27	32	37	42	47	52	27	0.5	29	72	11	82	88	93	86	10.3	108	118	128	143
		ii.	0.55	0.50	0.03	20.0	0.71	0.83	0.83	0.08	0.08	1.10	1.10	1.26	1.26	1.26	1.38	1.50	1.50	19.1	10.1	1.81	1.93	2.05	2.28
		mm.	14	15	91	17	18	21	21	25	25	25.5	28	32	32	32	35	38	38	41	41	46	40	52	28
		ii.	1 26	1.38	1.05	1.85	2.17	2.52	2.00	2.02	3.15	3.54	3.02	4.02	4.13	4.33	4.05	5.20	5.32	5.71	5.91	81.9	6.57	7.00	8.11
	၁	mm.	32	35	42	47	55	to.	9	7.5	8	8	92	102	105	110	118	132	135	145	150	157	167	180	506
	<u>m</u>	ii.	1.18	1.38	1.65	1.85	2.00	2.4.1	2.23	2.87	3.07	3.46	3.54	3.94	4.06	4.33	4.53	4.92	5.32	5.51	2.01	01.0	6.50	68.9	7.87
		mm.	30	35	42	47	53	62	+ 0	73	28	88	8	001	103	011	115	125	135	140	150	155	165	175	200
	A	in.	0.3937	0.5006	0.7874	0.9843	1.181.1	1.3780	1.5748	1.7717	1.9685	2.1654	2.3022	2.5951	2.7559	2.9528	3.1400	3.3465	3.5433	3.7402	3.9370	4.1339	4.5276	4.9213	5.5118
		mm.	OI	15	20	23	30	35	6	45	20	52	8	02	20	22	8	82	96	95	8	105	115	125	140
	Brg.	No.	1102	1103	1104	rios	9011	1107	2011	1100	OIII	IIII	1112	1113	1114	2111	OIII	LII1	8111	6III	1120	IIZI	1123	1125	1128



Bearing		Bore	Ū	Diameter	_	Width	Ba	Ball	Cham- fer	Inside	Ra-	Offset	Outside	Load
Š.	m/m	in.	m/m	.ii	m/m	in.	Diam.	No.	S	A	R	Q	B	R. P. M.
0204 0304 0404	20	.78740	47	1.85040	14 15	.55118	14.5 T	11 01	.05	1.0274	47,47	.0788	1.4704	200
0205 0305 0405	25	.98425	52 62 80	2.04725 2.44095 3.14962	15 17 21	.59055 .66929 .82677	യ ^{[ស} ്രധിതസിത	11 9 8	.o.	1.2243	မျာလ်မှာ အ <u>ကြ</u>	.0394 .0787 .0788	1.667 <i>2</i> 1.9884 2.5409	215 360 730
0206 0306 0406	30	01181.1	62 72 90	2.44095 2.83465 3.54332	16 19 23	.62992 .74803 .90551	2 1 1 1 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	10 8	.065	1.4711	2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	.0393	2.0809 2.3621 2.9148	260 530 925
0207 0307 0407	35	1.37795	72 80 100	2.83465 3.14962 3.93702	17 21 25	.66929 .82677 .98425	September 1991	112 10 8	.065	0899.1		.0394 .0788 .0788	2.3621 2.5209 3.3085	405 735 1150

COMBI	NED	RADI	AL A	ND TE	IKUSI	BEA	KINC	305
480 860 1375 625	1040	710 1280 1900	\$10 1500 2100	1025 1925 2400	1200 2275 2750	1325 2650	3000	3600
2.6571 2.8946 3.5257 2.8539	3.2883	3.0308 3.6620 4.2931	3.4245 3.8994 4.6868	3.6422 4.2731 5.0605	4.0357 4.6668 5.4542	4.2125 5.0405	4.4094 5.4342	4.7831 5.8079
.0394 .0787 .0787	.0788	.0394 .0787 .0787	.0394 .0788 .0787	.0394 .0787 .0788	.0393 .0787 .0787	.0394	.0394	.0393
က် <mark>လ</mark> လုလ က လုလ (၁၈) (၁၈) (၁၈) (၁၈) (၁၈) (၁၈) (၁၈) (၁၈)	2 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5	2 1 4 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	8 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5 5	8 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	1,0 10 10 14 14 14	6 4 1번 2	64 4	12 19 14 14
1.9148	2.1117	2.3385	2.5354	2.7622	2.9591	3.1816	3.3828	3.6097
80.	80.	0.85	.085	60.	ó.	.095	.095	oI.
41 00 41	o s	. 91	01 01	16 12 10	16 IO	18	10	16
www.min.min.min.min.min.min.min.min.min.min	njaor-jao d	www.dudu	τ <mark>π</mark> 9ε/4	1 1 6 1 1 6 1 1 6 1 1 6 1 1 6 1 1 6 1 1 6 1 1 1 6 1	H M M M M M M M M M M M M M M M M M M M	HIS THE	1 1 6	I Jews
.70866 .90551 1.06299	.98425	.78740 1.06299 1.22047	.82677 I.14173 I.29921	.86614 1.22047 1.37795	.90551 1.29921 1.45669	.94488	.98425	1.02362
18 23 27 19	25	20 27 31	21 29 33	22 31 35	23 33 37	24 35	37	39
3.14962 3.54332 4.33072 3.34647	3.93702	3.54332 4.33072 5.11813	3.93702 4.72443 5.51183	4.33072 5.11813 5.90554	4.72443 5.51183 6.29924	4.92128	5.11813 6.29924	5.51183
80 90 110 85	100	90	100 120 140	110 130 150	120 140 160	125 150	130	140 170
1.57481	1.77166	1.96851	2.16536	2.36221	2.55906	2.75591	2.95277	3.14962
04	55	50	55	99	65	70	7.5	8
0208	0300	0210 0310 0410	0211 0311 0411	0212 0312 0412	0213 0313 0413	0214	0215	0216

"SKF" SELF-ALIGNING RADIAL BEARINGS — MEDIUM TYPE — 13000

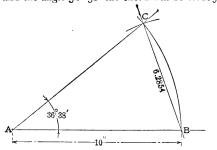
ute	00 1500	spar		230 200										_	2430 2150	_			_	_					
Revolutions per minute	500 1000	Maximum Load in Pounds	_	300	_	_		_	935 7			1000			3250 24			_	_	-				11200 87	-
Revolutio	300	aximum I	265	350	385	550	575	883	1100	1430	1760	2200	2540	3310	3860	4410	5070	5860	0000	8720	0016	10000	00111	13200	14300
J	150	M	310	410	405	999	685	1050	1320	1710	2120	2760	2980	3860	4410	5290	0200	7050	7280	10300	11000	13100	13600	c0091	17400
Ġ.		in.	.04	. 04	.o.	.04	o.	ō.	80.	80.	80.	80.	80.	80.	80.	.12	.12	.12	.12	.I2	.12	.I.2	.I2	.13	.12
Rad.		m/m	н	1	ı	н	Ħ	н	61	61	67	~	63	5	61	3	3	3	~	r	3	3	3	3	3
Width		in.	:	:	:	:	:	:	:	:	:	:	:	:	:	:	:	.:	:	1.929	2.047	2.1.46	2.283	2.362	2.520
Wic		m/m	:	:	:	:	:	:	:	:	:	:	:	:	:	:	:	:	:	49	52	54.5	258	09	† 9
Width	a	in,	.4331	.4724	.5118	.5512	.5006	.0093	.7480	.8268	.9055	.9843	1.0630	1.1417	1.2205	I.2992	I.3780	1.4567	I.5354	1.6142	1.6929	1.7717	1.8504	1.9291	1.9685
A		m/m	11	12	13	14	13	17	61	I.	23	25	. 27	29	31	33	35	37	39	41	43	45	47	49	20
Diameter	رر ا	ii.	1.3780	1.4267	I.0535	1.8504	2.0472	2.4400	2.8346	3.1496	3.5433	3.9370	4.3307	4.7244	5.1181	5.5118	5.9055	6.2992	6.6929	7.0866	7.4803	7.8740	8.4646	8.8583	9.4488
Dia		m/m	35	37	42	47	22	0.2	7.5	80	8	001	011	120	130	140	150	C91	170	180	061	200	215	225	240
Bore	5	ii.	.3937	.4724	.5000	.6693	-7874	.9843	1.181.1	I.3780	1.5748	1.7717	1.9685	2.1054	2.3622	2.5591	2.7559	2.9528	3.1496	3.3.465	3.5433	3.7.402	3.9370	4.1339	4.3307
-		m/m	OI	12	10	17	20	25	30	35	40	45	20	55	9	65	70	7.5	200	85	06	95	100	105	OII
	Type No.		1300	1301	1302	1303	1304	1305	1306	1307	1308	1300	1310	1311	1312	1313	1314	1315	1316	1317	1318	1319	1320	1321	1322
	BI							<u> </u>	^^	<=:			,		1	A TOP A			111111111111111111111111111111111111111		-	¥			

THE erection of a perpendicular by the construction of a triangle whose sides are respectively 3, 4 and 5 units in length is a familiar and handy device. The following table gives a greater range of choice in the shape or proportions of the triangle employed. The table is a list of all integral, or whole-number, right-angled triangles the units of whose least sides do not exceed 20.

Hight	Base	Hypot- enuse	Hight	Base	Hypot- enuse	Hight	Base	Hypot- enuse
3	4	5	12	16	20	17	144	195
5	12	13	12	35	37	18	24	30
6	8	10	13	84	37 85	18	8o	82
7	24	25	14	48	50	19	180	181
8	15	17	15	20	25	20	2 I	29
9	12	15	15	36	39	20	48	52
9	40	41	15	112	113	20	99	101
10	24	26	16	30	34			
II	60	61	16	63	65]		

TABLE OF CHORDS

To construct any angle from the table of chords, page 388: Let the required angle be 36° 38′; the nearest angles in the table are 36° 30′ and 36° 40′, and the chords are respectively 0.6263 and 0.6291, the difference 0.0028 corresponding to an angular difference of 10′. To find the amount which must be added to 0.6263 (the chord corresponding to 36° 30′) in order to obtain the chord for a 36° 38′ arc, multiply 0.0028 by $\frac{8}{10} = 0.00224$. 0.6263 + 0.00224 = 0.62854. Then, if the radius is 1″ and the angle 36° 38′ the chord will be 0.62854″.



In laying out an angle as in the accompanying illustration a base line A B can be drawn, say 10 inches long, then with a radius A B and center A, arc B C can be struck. Multiply chord 0.62854 inch by 10 giving 6.2854 inches, as the radius of an arc to be struck from center B and cutting arc B C at C. Through point C draw a line A C and the angle B A C will equal 36° 38°. Where the angle required is in even degrees or sixths of degrees

Where the angle required is in even degrees or sixths of degrees (as 10', 20', etc.) the corresponding chord may be taken directly from the table. A 10 to 1 layout is particularly convenient as the multiplication of the tabulated chords by 10 is readily performed mentally.

TABLE OF CHORDS THE TABULATED QUANTITIES = TWICE THE SINE OF HALF THE ARC

Deg.	o'	10'	20'	30'	40'	50'	60'
0	.0000	.0029	.0058	.0087	.0116	.0145	.0174
1	.0174	.0204	.0233	.0262	.0201	.0320	.0349
2	.0349	.0378	.0407	.0436	.0465	.0494	.0523
3	.0523	.0553	.0582	.0611	.0640	.0669	.0698
4	.0698	.0727	.0756	.0785	.0814	.0843	.0872
5 6	.0872	.0901	.0930	.0959	.0988	.1017	.1047
6	.1047	.1076	.1105	.1134	.1163	.1192	.1221
7 8	.1221	.1250	.1279	.1308	.1337	.1366	.1395
8	.1395	.1424	.1453	.1482	.1511	.1540	.1569
9	.1569	.1598	.1627	.1656	.1685	.1714	-1743
10	.1743	.1772	.1801	.1830	.1859	.1888	.1917
11	.1917	.1946	.1975	.2004	.2033	.2062	.2090
I 2	.2090	.2119	.2148	.2177	.2206	.2235	.2264
13	.2264	.2293	.2322	.2351	.2380	.2409	.2437
14	.2437	.2466	.2495	.2524	.2553	.2582	.2610
15	.2610	.2639	.2668	.2697	.2726	.2755	.2783
16	.2783	.2812	.2841	.2870	.2899	.2927	.2956
17	.2956	.2985	.3014	.3042	.3071	.3100	.3129
18	.3129	.3157	.3186	.3215	-3243	.3272	.3301
19	-3301	-3330	-3358	·33 ⁸ 7	.3416	•3444	•3473
20	·3473	.3502	·353°	·3559	.3587	.3616	.3645
21	.3645	·3673	.3702	.3730	•3759	.3788	.3816
22	.3816	.3845	.3873	.3902	•3930	•3959	.3987
23	.3987	.4016	.4044	.4073	.4101	.4130	.4158
24	.4158	.4187	.4215	.4243	.4272	.4300	•4329
25 26	•4329	•4357	.4385	.4414	•4442	·4471	•4499
	•4499	·4527	.4556	.4584	.4612	.4641	.4669
27 28	.4669	.4697	.4725	·4754	.4782	.4810	.4838
	.4838	.4867	.4895	•4923	.4951	•4979	.5008
29	.5008	.5036	.5064	.5092	.5120	.5148	.5176
30	.5176	.5204	.5232	.5261	.5289	•5317	•5345
31	•5345	•5373	.5401	•5429	•5457	.5485	.5513 .5680
32	.5513	.5541	.5569	.5596	.5624	.5652 .5820	.5000
33	.5680 .5847	.5708 .5875	.5736	.5764	.5792		.5847 .6014
34	.6014	.6042	.5903 .6069	.5931 .6097	.5959 .6125	.5986 .6153	.6180
35	.6180	.6208	.6236	.6263	.6291	.6318	.6346
36	.6346	.6374	.6401	.6429	.6456	.6484	.6511
37 38	.6511	.6539	.6566	.6594	.6621	.6649	.6676
39	.6676	.6703	.6731	.6758	.6786	.6813	.6840
39 40	.6840	.6868	.6895	.6922	.6950	.6977	.7004
41	.7004	.7031	.7°59	.7086	.7113	.7140	.7167
42	.7167	.7194	.7059	·7249	.7276	.7303	.7330
43	.7330	·7357	.7384	.7411	.7438	.7465	.7492
44	·733°	.7519	.7546	.7573	.7600	.7627	.7654
45	.7654	.7680	.7707	•7734	.7761	.7788	.7815

TABLE OF CHORDS

THE TABULATED QUANTITIES = TWICE THE SINE OF HALF THE ARC

Deg.	0'	10'	20′	30'	40′	50'	60'
46	.7815	.7841	.7868	.7895	.7921	.7948	•7975
47	.7975	.8001	.8028	.8055	.8081	.8108	.8135
48	.8135	.8161	.8188	.8214	.8241	.8267	.8294
49	.8294	.8320	.8347	.8373	.8400	.8426	.8452
50	.8452	.8479	.8505	.8531	.8558	.8584	.8610
	.8610	.8636	.8663	.8689	.8715	.8741	.8767
51	.8767	.8793	.8820	.8846	.8872	.8898	.8924
52	.8924	.8950	.8976	.9002	.9028		.9080
53	.9080	.9106			.9028	.9054	
54		.9261	.9132 .9286	.9157	.9338	.9209	.9235
55	.9235			.9312 .9466		.9364	.9389
56	.9389	.9415	.9441		.9492	.9518	.9543
57	.9543	.9569	•9594	.9620	.9645	.9671	.9696
58	.9696	.9722	•9747	.9772	.9798	.9823	.9848
59	.9848	.9874	.9899	.9924	•9949	•9975	1.0000
60	1.0000	1.0025	1.0050	1.0075	1.0100	1.0126	1.0151
61	1.0151	1.0176	1.0201	1.0226	1.0251	1.0276	1.0301
62	1.0301	1.0326	1.0350	1.0375	1.0400	1.0425	1.0450
63	1.0450	1.0475	1.0500	1.0524	1.0550	1.0574	1.0598
64	1.0598	1.0623	1.0648	1.0672	1.0697	1.0721	1.0746
65	1.0746	1.0770	1.0795	1.0819	1.0844	1.0868	1.0893
66	1.0893	1.0917	1.0941	1.0966	1.0990	1.1014	1.1039
67	1.1039	1.1063	1.1087	1.1111	1.1135	1.1159	1.1184
68	1.1184	1.1208	1.1232	1.1256	1.1280	1.1304	1.1328
69	1.1328	1.1352	1.1376	1.1400	1.1424	1.1448	1.1471
70	1.1471	1.1495	1.1519	1.1543	1.1567	1.1590	1.1614
71	1.1614	1.1638	1.1661	1.1685	1.1708	1.1732	1.1756
72	1.1756	1.1780	1.1803	1.1826	1.1850	1.1873	1.1896
73	1.1896	1.1920	1.1943	1.1966	1.1990	1.2013	1.2036
74	1.2036	1.2059	1.2083	1.2106	1.2129	1.2152	1.2175
75	1.2175	1.2198	1.2221	1.2244	1.2267	1.2290	1.2313
76	1.2313	1.2336	1.2360	1.2382	1.2405	1.2427	1.2450
77 78	1.2450	1.2473	1.2496	1.2518	1.2541	1.2564	1.2586
	1.2586	1.2609	1.2631	1.2654	1.2677	1.2699	1.2721
79	1.2721	1.2744	1.2766	1.2789	1.2811	1.2833	1.2856
80	1.2856	1.2878	1.2900	1.2922	1.2945	1.2967	1.2989
81	1.2989	1.3011	1.3033	1.3055	1.3077	1.3099	1.3121
82	1.3121	1.3143	1.3165	1.3187	1.3209	1.3231	1.3252
83	1.3252	1.3274	1.3296	1.3318	1.3340	1.3361	1.3383
84	1.3383	1.3404	1.3426	1.3447	1.3469	1.3490	1.3512
85	1.3512	1.3533	1.3555	1.3576	1.3597	1.3619	1.3640
86	1.3640	1.3661	1.3682	1.3704	1.3725	1.3746	1.3767
87	1.3767	1.3788	1.3809	1.3030	1.3851	1.3872	1.3893
88	1.3893	1.3914	1.3935	1.3956	1.3977	1.3997	1.4018
89	1.4018	1.4039	1.4060	1.4080	1.4101	1.4121	1.4142
90	1.4142		"				

TABLE FOR SPACING HOLES IN CIRCLES

No. of Divisions in Circle	Deg. of Arc	Length of Chord Dia. 1	Length of Chord Dia. 2	Length of Chord Dia. 3	Length of Chord Dia. 4	Length of Chord Dia. 5	Length of Chord Dia. 6
3 4 5 6 7 8 9 10 11 12 13 14	120 90 72 60 51°-25' 45 40 36 32°-43' 30 27°-41' 25°-42'	.866 .707 .588 .500 .434 .383 .342 .309 .282 .259	1.732 1.414 1.176 1.000 .868 .765 .684 .618 .564 .518	2.598 2.121 1.763 1.500 1.302 1.148 1.026 .927 .845 .776 .718	3.464 2.828 2.351 2.000 1.736 1.531 1.368 1.236 1.127 1.035	4.330 3.536 2.938 2.500 2.170 1.913 1.710 1.545 1.409 1.294 1.197	5.196 4.243 3.527 3.000 2.604 2.296 2.052 1.854 1.691 1.553 1.436 1.334
15 16 17	24 22°-30′ 21°-11′	.208 .195 .184	.416	.624 .585	.832 .780 .735	1.040 .975 .918	1.247 1.171 1.102
18 19 20 21 22	20 18°-57' 18 17°-8' 16°-22'	.174 .164 .156 .149	.347 .329 .318 .298	.521 .493 .469 .447 .427	.695 .658 .626 .596	.868 .822 .782 .745	.987 .937 .894 .855
23 24 25 26 27	15°-39' 15 14°-24' 13°-51' 13°-20'	.136 .130 .125 .120	.273 .261 .251 .241	.409 .392 .375 .361 .348	.545 .522 .501 .482 .464	.681 .653 .627 .602	.818 .783 .752 .723 .697
28 29 30 31 32	12°-51' 12°-25' 12 11°-37' 11°-15'	.112 .108 .104 .101	.224 .216 .209 .202	.336 .324 .314 .303 .294	.448 .432 .418 404 .393	.560 .540 .522 .505 .491	.672 .648 .627 .606

TABLE FOR SPACING HOLES IN CIRCLES

No. of Divisions in Circle	Deg. of Arc	Length of Chord Dia. 7	Length of Chord Dia. 8	Length of Chord Dia. 9	Length of Chord	Length of Chord Dia. 11	Length of Chord Dia. 12
3 4 5 6 7 8 9 10	120 90 72 60 51°-25' 45 40 36 32°-43'	6.062 4.950 4.115 3.500 3.037 2.679 2.394 2.163 1.973	6.928 5.657 4.702 4.000 3.471 3.061 2.736 2.472	7.794 6.364 5.290 4.500 3.905 3.444 3.078 2.781 2.536	8.660 7.071 5.878 5.000 4.339 3.827 3.420 3.090 2.818	9.526 7.778 6.465 5.500 4.773 4.210 3.762 3.399 3.100	10.392 8.485 7.053 6.000 5.207 4.592 4.104 3.708 3.381
12	30 30	1.812	2.069	2.329	2.588	2.847	3.106
13 14 15 16 17 18 19	27°-41′ 25°-42′ 24 22°-30′ 21°-11′ 20 18°-57′ 18	1.676 1.557 1.455 1.366 1.286 1.216 1.151 1.095	1.915 1.779 1.663 1.561 1.469 1.389 1.316	2.154 2.000 1.871 1.756 1.653 1.563 1.480	2.394 2.224 2.079 1.951 1.837 1.737 1.645	2.633 2.446 2.287 2.146 2.020 1.910 1.809 1.721	2.873 2.669 2.495 2.341 2.204 2.084 1.974 1.877
21	17°- 8′	1.043	1.192	1.341	1.489	1.639	1.788
22	160-22'	.996	1.139	1.281	1.423	1.566	1.708
23 24 25 26 27	15°-39' 15 14°-24' 13°-51' 13°-20'	.954 .914 .877 .843 .813	1.092 1.044 1.003 .963	1.227 1.175 1.128 1.084 1.045	1.363 1.305 1.253 1.204 1.161	1.499 1.436 1.379 1.325 1.277	1.635 1.566 1.504 1.445 1.393
28 2 9	12°-51' 12°-25'	.784 .756	.896 .864	1.008 •972	1.121	1.233 1.188	1.345 1.296
30	12	.732	.836	.941	1.045	1.150	1.254
31 32	11°-37′ 11°-15′	.707 .687	.808 .785	.910 .883	1.011	1.112	1.213
	!	1				·	1

TABLE FOR SPACING HOLES IN CIRCLES

The table on pages 390 and 391 will be found of service when it is desired to space any number of holes up to and including 32, in a circle. The number of divisions or holes desired will be found in the first column, the corresponding angle included at the center being given as a convenience in the second column. The remaining column heads cover various diameters of circles from 1 to 12 inches, and under these different heads and opposite the required number of holes will be found the lengths of chords or distances between hole centers for the given circle diameter.

Thus, if it is required to space off 18 holes in an 8-inch circle, by following down the first column until 18 is reached and then reading directly to the right, in the column headed "Length of Chord-Dia. 8," will be found the distance 1.380 as the chord length for that number of divisions and diameter of circle. Or, suppose a circle of 12 inches diameter is to be spaced off for a series of 27 holes to be drilled at equal distances apart: Opposite 27 found in the first column, and under the heading, "Dia. 12," will be found the chord 1.303 as the length to which the dividers may be set directly for lay-

ing off the series of holes.

If it is desired to lay off a series of holes in a circle of some diameter not given in the table, say 10 holes in an 11½-inch circle, subtract the chord for 10 holes in an 11-inch circle, or, 3.399 from the chord in the "Dia. 12" column, or 3.708, and add half the difference (.154) to 3.309, giving 3.553 as the chord or center distance between holes. Or, if 24 holes are to be equally spaced in a 20-inch circle, all that is necessary in order to find the chord, or center distance, is to find opposite 24, and in the column headed, "Dia. 10," the quantity 1.305 and multiply this by 2, giving a length of 2.610 inches as the center distance.

TABLE OF SIDES, ANGLES AND SINES

The table on pages 393 to 397 is carried out for a much higher number of sides or spaces than are included in the preceding table and will be found useful in many cases not covered by that table. It was originally computed for finding the thicknesses of commutator bars and also for calculating the chord for spacing slots in armature punchings. In using this table the diameter of the circle is, of course, multiplied by the sine opposite the desired number of holes or sides.

Assuming for illustration that a series of 51 holes are to be equally spaced about a circle having a diameter of 17 inches, opposite 51 in the column headed "No. of Sides," find the quantity .06156 in the column headed "Sine," and multiply this quantity by 17. The product 1.0456 is the length of the chord or the required distance between centers of the holes for this circle. Or, if 40 equidistant points are to be spaced about a circle 16 inches diameter, opposite the number of sides, 40, will be found the quantity .078459 which multiplied by 16 gives 1.255 inch as the distance between centers.

	(Angle)	given is half of a	ngie su	otended at center,	
No. Sides	Angle Deg. Min. Sec.	Sine	No. Sides	Angle Deg. Min. Sec.	Sine
. 3	60	.8660254	52	3-27-41.53	.0603784
4	45	.7071067	53	3-23-46.41	0592405
5	36	.5877852	54	3-20	.0581448
5 6	30	.5000000	55	3-16-21.81	.0570887
	25-42-51.42	.4338828	56	3-12-51.42	.0560704
7 8	22-30	.3826834	57	3- 9-28.42	.0550877
9	20-	.3420201	58	3- 6-12.41	.0541388
10	18-	.3090170	59	3- 3- 3.05	.0532221
ΙI	16-21-49.09	.2817325	60	3-	.0523360
12	15-	.2588190	61	2-57- 2.95	.0514787
13	13-50-46.15	.2393157	62	2-54-11.61	.0506491
14	12-51-25.71	.2225208	63	2-51-25.71	.0498458
15	12	.2079116	64	2-48- 45	.0490676
16	11-15	.1950903	65	2-46- 9.23	.0483133
17	10-35-17.64	.1837495	66	2-43-38.18	.0475819
18	10-	.1736481	67	2-41-11.64	.0468722
19	9-28-25.26	.1645945	68	2-38-49.41	.0461834
20	ý-	.1564344	69	2-36-31.30	.0455145
21	8-34-17.14	.1490422	70	2-34-17.14	.0448648
22	8-10-54.54	.1423148	71	2-32- 6.76	.0442333
23	7-49-33.91	.1361666	72	2-30	.0436194
24	7-30-	.1 305262	73	2-27-56.71	.0430272
25	7-12-	.1253332	74	2-25-56.75	.0424411
26	6-55-23.07	.1205366	75	2-24-	.0418757
27	6-40	.1160929	76	2-22- 6.31	.0413249
28	6-25-42.85	.1119644	77	2-20-15.58	.0407885
29	6-12-24.82	.1081180	78	2-18-27.60	.0402659
30	6-	.1045284	79	2-16-42.53	.0397565
31	5-48-23.22	.1011683	80	2-15-	.0392598
32	5-37-30	.0980171	81	2-13-20	.0387753
33	5-27-16.36	.0950560	82	2-11-42.45	.0383027
34	5-17-38.82	.0922683	83	2-10- 7.22	.0378414
35	5- 8-34.28	.0896392	84	2- 8-34.28	.0373911
36	5-	.0871557	85	2- 7- 3.54	.0369515
37	4-51-53.51	.0848058	86	2- 5-34.88	.0365220
3 8	4-44-12.63	.0825793	87	2- 4- 8.27	.0361023
3 9	4-36-55.38	.0804665	88	2- 2-43.63	.0356923
40	4-30-	.0784591	89	2- 1-20.89	.0352914
41	4-23-24.87	.0765492	90	2-	.0348995
42	4-17-8.57	.0747301	91	1-58-40.87	.0345160
43	4-11- 9.76	.0729952	92	1-57-23.47	.0341410
44	4- 5-27.27	.0713391	93	1-56- 7.74	.0337741
45	4	.0697565	94	1-54-53.61	.0334149
46	3-54-46.95	.0682423	95	1-53-41.05	.0330633
47	3-49-47-23	.0667926	96	1-52-30.	.0327190
48	3-45-	.0654031	97	1-51-20.41	.0323818
49	3-40-24.49	.0640702	98	1-50-12.24	.0320515
50	3-36-	.0627905	99	1-49- 5.45	.0317279
_51	3-31-45.88	.06156c9	100	1-48-	.0314107

TABLE OF SIDES, ANGLES AND SINES

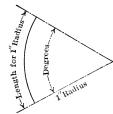
No. Angle Deg. Min. Sec. Sine Sides Deg. Min. Sec.	Sine
101 1-46-55.84 .0310998 151 1-11-31.39	.0208037
102 1-45-52.94 .0307950 152 1-11- 3.15	.0206668
103 1-44-51.26 .0304961 153 1-10-35.29	.0205318
104 1-43-50.76 .0302029 154 1-10- 7.79	.0203985
105 1-42-51.42 .0299154 155 1- 9-40.64	.0202669
106 1-41-53.20 .0296332 156 1-9-13.84	.0201370
107 1-40-56.07 .0293564 157 1-8-47.38	.0200087
108 1-40- .0290847 158 1-8-21.26	.0198821
109 1-39- 4.95 .0288179 159 1- 7-55.47	.0197571
110 1-38-10.90 .0285560 160 1-7-30	.0196336
111 1-37-17.83 .0282488 161 1- 7- 4.84	.0195117
112 1-36-25.71 .0280462 162 1-6-40	.0193913
113 1-35-34.51 .0277981 163 1-6-15.46	.0192723
114 1-34-44.21 .0275543 164 1- 5-51.21	.0191548
115 1-33-54.78 .0273147 165 1- 5-27.27	.0190387
116 1-33- 6.20 .0270793 166 1- 5- 3.61	.0189241
117 1-32-18.46 .0268479 167 1- 4-40.23	.0188107
118 1-31-31.52 .0266204 168 1- 4-17.14	.0186988
119 1-30-45.38 .0263968 169 1- 3-54.31	.0185881
120 1-300261769 170 1-3-31.76	.0184788
121 1-29-15.37 .0259606 171 1- 3- 9.47	.0183708
122 1-28-31.47 .0257478 172 1- 2-47.44	.0182640
123 1-27-48.29 .0255386 173 1-2-25.66	.0181584
124 1-27- 5.80 .0253326 174 1- 2- 4.13	.0180541
125 1-26-24 .0251300 175 1-1-42.85	.0179509
126 1-25-42.85 .0249306 176 1- 1-21.81	.0178489
127 1-25- 2.36 .0247344 177 1- 1- 1.01	.0177481
128 1-24-22.50 .0245412 178 1- 0-40.44	.0176484
129 1-23-43.25 .0243509 179 1- 0-20.11	.0175498
130 1-23- 4.61 .0241637 180 1	.0174524
131 1-22-26.56 .0239793 181 -59-40.11	.0173559
132 1-21-49.09 .0237976 182 -59-20.43	.0172605
133 1-21-12.18 .0236188 183 -59- 0.98	.0171663
134 1-20-35.82 .0234425 184 -58-41.73	.0170730
135 1-20- .0232689 185 -58-22.70	.0169807
136 1-19-24.70 .0230978 186 -58- 3.87	.0168894
137 1-18-49.92 .0229292 187 -57-45.24	.0167991
138 1-18-15.65 .0227631 188 -57-26.30	.0167097
139 1-17-41.87 .0225994 189 -57- 8.57	.0166214
140 1-17- 8.57 .0224380 190 -56-50.52	.0165339
141 1-16-35.74 .0222789 191 -56-32.67	.0164473
142 1-16- 3.38 .0221220 192 -56-15	.0163617
143 1-15-31.46 .0219673 193 -55-57.51	.0162769
144 1-150218148 194 -55-40.20	.0161930
145 1-14-28.96 .0216644 195 -55-23.07	.0161100
146 1-13-58.35 .0215160 196 -55-6.12	.0160278
147 1-13-28.16 .0213697 197 -54-49.34	.0159464
148 1-12-58.37 .0212253 198 -54-32.72	.0158659
149 1-12-28.99 0210829 199 -54-16.28	.0157862
150 1-12- .0209424 200 -54-	.0157073

No. Sides	Angle Min. Sec.	Sine	No. Sides	Angle Min. Sec.	Sine
201	53-43.88	.0156244	251	43- 1.67	.0125160
202	53-27.92	.0155518	252	42-51.43	.0124663
203	53-12.12	.0154752	253	42-41.26	.0124171
204	52-56.47	.0153993	254	42-31.18	.0123682
205	52-40.97	.0153242	255	42-21.18	.0123197
206	52-25.63	.0152498	256	42-11.25	.0122715
207	52-10.44	.0151764	257	42- 1.40	.0122238
208	51-55.38	.0151033	258	41-51.63	.0121764
209	51-40.48	.0150310	259	41-41.93	.0121294
210	51-25.71	.0149595	260	41-32.31	.0120827
2 I I	51-11.00	.0148886	261	41-22.76	.0120364
212	50-56.60	.0148183	262	41-13.28	.0119905
213	50-42.25	.0147487	263	41- 3.88	.0119449
214	50-28.04	.0146798	264	40-54.54	.0118997
215	50-13.96	.0146115	265	40-45.28	.0118548
216	50-	.0145439	266	40-36.09	.0118102
217	49-46.17	.0144769	267	40-26.96	.0117660
218	49-32.48	.0144104	268	40-17.91	.0117221
219	49-18.91	.0143446	269	40- 8.93	.0116786
220	49- 5.46	.0142794	270	40→	.0116353
22 I	48-52.13	.0142148	271	39-51.14	.0115923
222	48-38.92	.0141508	272	39-42.35	.0115497
223	48-25.83	.0140874	273	39-33.63	0115074
224	48-12.86	.0140245	274	39-24.96	.0114654
225	48-	.0139622	275	39-16.36	.0114237
226	47-47.26	.0139004	276	39∸ 7.83	.0113823
227	47-34.63	.0138392	277	38-59.35	.0113412
228	47-22.11	.0137785	278	38-50.94	.0113004
229	47- 9.69	.0137183	279	38-42.58	.0112599
230	46-57.39	.0136587	280	38-34.28	.0112197
231	46-45.19	.0135995	281	38-26.05	.0111798
232	46-33.10	.0135409	282	38-17.87	.0111401
233	46-21.11	.0134828	283	38- 9.75	.0111008
234	46- 9.23	.0134252	284	38- 1.69	.0110617
235	45-57.45	.0133681	285	37-53.68	.0110229
236	45-45.76	.0133115	286	17-45.73	.0109844
237	45-34.18	.0132553	287	37-37.84	.0109461
238	45-22.69	.0131996	288	37-30	.0109081
239	45-11.29	.0131444	289	37-22.21	.0108704
240	45-	.0130896	290	37-14.48	0108329
241	44-48.80	.0130353	291	37- 6.80	.0107957
242	44-37.68	.0129814	292	36-59.18	.0107587
243	44-26.67	.0120280	293	36-51.60	.0107220
244	44-15.74	.0128750	294	36-44.08	
245	44- 4.90	.0128225	295	36-36.61	.0106493
246	43-54.15	.0127704	296	36-29.19	.0106133
247	43-43.48	.0127187	297	36-21.82	.0105776
248	43-32.40	.0126674	298	36-14.50	.0105421
249 · 250		.0126165	299	36- 7.22	.0105068
250	43-12	.0125661	300	36-	,0104/10

TABLE OF SIDES, ANGLES AND SINES

No. Sides	Angle Min. Sec.	Sine	No. Sides	Angle Min. Sec.	Sine
301	35-52.82	.0104370	351	30-46.15	.0089502
302	3 5-45.69	0104024	352	30-40.91	.0089248
303	35-38.61	.0103681	353	30-35.69	.0088996
304	35-31.58	.0103340	354	30-30.51	.0088744
305	35-24.59	.0103001	355	30-25.35	.0088494
306	35-17.65	.0102665	356	30-20.22	.0088245
307	35-10.75	.0102330	357	30-15.12	.0087998
308	35- 3.90	.0101998	358	30-10.05	.0087753
309	34-57.09	.0101668	359	30- 5.01	.0087508
310	34-50.32	.0101340	360	30-	.0087265
311	34-43.60	.0101014	361	29-55.01	.0087023
312	34-36.92	.0100690	362	29-50.05	.0086783
313	34-30.29	.0100368	363	29-45.12	.0086544
314	34-23.69	.0100049	364	29-40.22	.0086306
315	34-17.14	.0099731	365	29-35.34	.0086070
316	34-10.63	.0099415	366	29-30.49	.0085835
317	34- 4.16	.0099102	367	29-25.67	.0085601
318	33-57.74	.0098791	368	29-20.87	.0085368
319	33-51.35	.0098482	369	29-16.10	.0085137
320	33-45	.0098174	370	29-11.35	.0084907
321	33-38.69	.0097868	371	29- 6.63	.0084678
322	33-32.42	.0097564	372	29- 1.94	.0084451
323	33-26.19	.0097261	373	28-57.27	.0084224
324	33-20	.0096961	374	28-52.62	.0083999
325	33-13.85	.0096663	375	28-48	.0083775
326	33- 7.73	•.0096367	376	28-43.40	.0083552.
327	33- 1.65	.0096072	377	28-38.83	.0083331
328	32-55.61	.0095779	378	28-34.28	.0083110
329	32-49.60	.0095488	379	28-29.76	.0082891
330	32-43.64	.0095198	380	28-25.26	.0082673
331	32-37.70	.0094911	381	28-20.78	.0082456
332	32-31.81	.0094625	382	28-16.33	.0082240
333	32-25.95	.0094341	383	28-11.91	.0082025
334	32-20.12	.0094059	384	28- 7.50	.0081812
335	32-14.33	.0093778	385	28- 3.12	.0081599
336	32- 8.57	.0093499	386	27-58.76	.0081387
337	32- 2.85	.0093221	387	27-54.42	.0081177
338	31-57.16	.0092945	388	27-50.10	.0080968
339	31-51.50	.0092671	389	27-45.81	.0080760
340	31-45.88	.0092398	390	27-41.54	.0080553
341	31-40.29	.0092127	391	27-37.29	.0080347
342	31-34.74	.0091858	392	27-33.06	.0080142
343	31-29.21	.0091590	393	27-28.85	.0079938
344	31-23.72	.0091324	394	27-24.67	.0079735
345	31-18.26	.0091059	395	27-20.51	.0079533
346	31-12.83	.0090796	396	27-16.36	.0079332
347 348	31- 7.44	.0090534	397	27-12.24	.0079132
	31- 2.07	.0090274	398	27-8.14	.0078934
349 35 0	30-56.73	.0000016	399	27- 4.06	.0078736
350	30-51.43	.0 089758	400	27-	.0078534

		iven is nan or at		rended at center	/
No. Sides	Angle Min. Sec.	Sine	No. Sides	Angle Min. Sec.	Sine
401	26-55.96	.0078343	451	23-56.81	.0069658
402	26-51.94	.0078148	452	23-53.63	.0069504
403	26-47.94	.0077954	453	23-50.46	.0069351
404	26-43.96	.0077761	454	23-47.31	.0069198
405	26-40	.0077569	455	23-44.17	.0069046
406	26-36.06	.0077378	456	23-41.05	.0068894
407	26-32.14	.0077188	457	23-37.94	.0068744
408	26-28.23	.0076999	458	23-34.84	.0068594
409	26-24.35	.0076811	459	23-31.76	.0068444
410	26-20.49	.0076623	460	23-28.69	.0068295
411	26-16.64	.0076437	461	23-25.64	.0068147
412	26-12.82	.0076251	462	23-22.60	.0067999
413	26- 9.01	.0076067	463	23-19.57	.0067852
414	26- 5.22	.0075883	464	23-16.55	.0067706
415	26- 1.45	.0075700	465	23-13.55	.0067561
416	25-57.70	.0075518	466	23-10.56	.0067416
417	25-53.96	.0075337	467	23- 7.58	.0067272
418	25-50.24	.0075157	468	23- 4.61	.0067128
419	25-46.54	.0074977	469	23- 1.66	.0066985
420	25-42.86	.0074799	470	22-58.72	.0066842
421	25-39.19	.0074621	471	22-55.79	.0066700
422	25-35.54	.0074444	472	22-52.88	.0066559
423	25-31.91	.0074268	473	22-49.98	.0066418
424	25-28.30	.0074093	474	22-47.09	.0066278
425	25-24.70	.0073919	475	22-44.21	.0066138
426	25-21.12	.0073745	476	22-41.34	.0065999
427	25-17.56	.0073573	477	22-38.49	.0065861
428	25-14.02	.0073401	478	22-35.65	.0065723
429	25-10.49	.0073230	479	22-32.82	.0065585
430	25- 6.98	.0073059	480 481	22-30	.0065449
431	25- 3.48	.0072890		22-27.20	.0065313
432	25-	.0072721	482	22-24.40	.0065178
433	24-56.54	.0072553	484	22-21.61	.0065043
434	24-53.09 24-49.66	.0072386	485	22-16.08	.0064909
435 436	24-46.24	.0072220	486		.0064775
437	24-40.24	.0072054	487	22-13.33	.0064509
437	24-39.45	.0071725	488	22-7.87	.0004377
439	24-39.45	.0071725	489	22- 7.37	.0064245
440	24-32.73	.0071302	490	22- 3.10	.0064114
441	24-29.39	.0071237	491	21-59.75	.0063983
442	24-26.06	.0071237	491	21-57.07	.0063853
443	24-22.75	.0070916	493	21-54.40	.0063723
444	24-19.46	.0070756	493	21-51.74	.0063594
445	24-16.18	.0070597	495	21-49.09	.0063466
446	24-12.91	.0070439	495	21-46.45	.0063338
447	24- 9.66	.0070281	497	21-43.82	.0063211
448	24- 6.43	.0070124	498	21-41.20	.0063084
449	24- 3.21	.0069968	499	21-38.59	.0062957
450	24-	.0069813	500	21-36	.0062831
	•		5-5	5-	, ,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,,



LENGTHS OF CIRCULAR ARCS

THE table gives the lengths of circular arcs to the radius of one, for angles from 1 to 180 degrees. The lengths for minutes of

arcs are given at the right.

To find the length of a circular arc with radius of 1 inch and angle of 45 degrees 20 minutes. Opposite 45 degrees find 0.7854, and opposite 20 minutes 0.0058. Adding these gives 0.7012 inch as the length of arc.

If the radius is 2 inches, multiply the lengths in the table by 2.

		LEN	GTHS O	F CI	RCULAR	ARC	s то Б	CADIU	JS OF 1	[
De- gree	Length	De- gree	Length	De- gree	Length	De- gree	Length	Min.	Length	Min.	Length
0	0.0000	45	0.7854	90	1.5708	135	2.3562	0	0.0000	45	0.0131
I	0.0175	46	0.8020	QI	1.5882	136	2.3736	1	0.0003	46	0.0134
2	0.0349	47	0.8203	02	1.6057	137	2.3011	2	0.0006	47	0.0137
3	0.0524	48	0.8378	9,3	1.6232	138	2.4086	3	0.0000	48	0.0140
4	0.0698	19	0.8552	94	1.6406	139	2.4260	4	0.0012	49	0.0143
- 7	0.0873	50	0.8728	95	1.6581	140	2.4435	5	0.0015	50	0.0145
5 6	0.10.17	51	0.8901	96	1.6755	1.11	2.4600	5 6	0.0017	51	0.0148
7	0.1222	52	0.9076	97	1.6930	142	2.4784	7	0.0020	52	0.0151
8	0.1306	53	0.9250	98	1.7104	143	2.4958	7 8	0.0023	53	0.0154
Q	0.1571	54	0.9425	99	1.7279	144	2.5133	9	0.0026	54	0.0157
10	0.1745	55	0.9599	100	1.7453	145	2.5307	10	0.0029	5.5	0.0160
11	0.1020	56	0.9774	101	1.7628	146	2.5482	II	0.0032	56	0.0163
12	0.2004	57	0.0048	102	1.7802	147	2.5656	12	0.0035	57	0.0166
13	0.2260	58	1.0123	103	1.7977	148	2.5831	13	0.0038	58	0.0169
14	0.2443	59	1.0297	104	1.8151	149	2.0005	14	0.0041	59	0.0172
15	0.2618	60	1.0472	105	1.8326	150	2.6180	15	0.0044	60	0.0175
16	0.2793	61	1.0647	106	1.8500	151	2.6354	16	0.0047	1	
17	0.2967	62	1.0821	107	1.8675	152	2.6529	17	0.0050	1	1
18	0.3142	63	1.0996	108	1.8850	153	2.6704	18	0.0052		
19	0.3316	64	1.1170	100	1.9024	154	2.6878	19	0.0055	1	
20	0.3491	65	1.1345	110	1.9199	155	2.7052	20	0.0058		ŀ
2 I	0.3665	66	1.1519	III	1.9373	156	2.7227	2 I	0.0061	1	ŀ
22	0.3840	67	1.1694	112	1.9548	157	2.7.402	22	0.006.1		
23	0.4014	68	1.1868	113	1.9722	158	2.7576	23	0.0067		
24	0.4189	69	1.2043	114	1.9897	159	2.775I	24	0.0070	1	
25	0.4363	70	1.2217	115	2.007 I	160	2.7925	25	0.0073		1
26	0.4538	71	1.2392	116	2.0246	161	2.8100	26	0.0076		
27	0.4712	72	1.2566	117	2.0420	162	2.8274	27	0.0079	1	
28	0.4887	73	1.2741	118	2.0595	163	2.8449	28	0.0081	1	1
29	0.5061	74	1.2915	119	2.0769	164	2.8623	29	0.0084		
30	0.5236	7.5	1.3000	120	2.0944	165	2.8798	30	0.0087	1	1
31	0.5411	76	1.3265	121	2.1118	166	2.8972	31	0.0000		
32	0.5585	77	1.3439	122	2.1293	167	2.9147	32	0.0093		
33	0.5760	78	1.3614	123	2.1468	108	2.9322	33	0.0096		
34	0.5934	79	1.3788	124	2.1642	169	2.9496	34	0.0099	1	
35	0.6100	80	1.3963	125	2.1817	170	2.9671	35	0.0102		
36	0.6283	81	1.4137	126	2.1991	171	2.9845	36	0.0105		į .
37	0.6458	82	1.4312	127	2.2166	172	3.0020	37	0.0108	1	1
38	0.6632	83	1.4486	128	2.2340	173	3.0104	38	0.0111	1	
39	0.6807	84	1.4661	129	2.2515	17/4	3.0369	39	0.0113		
40	0.6981	85	1.4835	130	2.2690	175	3.0543	40	0.0116		
41	0.7156	86	1.5010	131	2.2864	176	3.0718	41	0.0119		
42	0.7330	87	1.5184	132	2.3038	177	3.0892	42	0.0122	1	
43	0.7505	88	1.5359	133	2.3132	178	3.1007	43	0.0125		
44	0.7679	89	1.5533	134	2.3387	179	3.1241	44	0.0128	1	i

'ACTUAL CUTTING SPEED OF PLANERS IN FEET PER MINUTE

Forward Cut- ting Speed	Return Speed										
in Feet per Minute	2 to 1	3 to 1	4 to 1	5 to 1	6 to 1	7 to 1	8 to 1				
20	13.3	15.	16	16.66	17.14	17.5	17.76				
25	16.6	18.75	20	20.83	21.42	21.87	22.16				
30	20.	22.5	2.4	25.	25.71	26.25	26.56				
35	23.3	26.25	28	29.16	30.	30.62	31.04				
40	26.6	30.	32	33.33	34.28	35.	35.52				
45	30.	33.75	36	37 5	38.56	39.37	40.				
50	33.3	37.5	40	41.66	42.84	43.75	44.48				
55	36.6	41.25	44	45.83	47.12	48.12	48.95				
60	40.	45.	48	50.	51.42	52.50	53.43				
65	43.3	48.75	52	54.16	55.70	56.87	57.91				
70	46.6	52.5	56	58.33	60.	61.25	62.3				
75	50.	56.25	60	62.5	64.28	66,62	66.71				

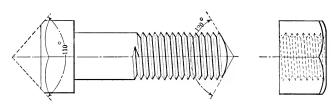
The table shows clearly that a slight increase in cutting speed is better than high return speed. A 25-foot forward speed at 4 to 1 return is much better than 8 to 1 return with 20-feet forward speed. Economical planer speeds are given below (Cincinnati Planer Co.).

Cast Iron roughing.....40 to 50 ft.; finishing.....20 to 25 ft. Steel casting and wrought iron roughing 30 to 35 ft.; finishing 20 ft. Bronze and brass...50 to 60 ft.; Machinery steel...30 to 35 ft.

ALLOWANCES FOR BOLT HEADS AND UPSETS

STOCK ALLOWED FOR STANDARD UPSETS BY ACME MACHINERY CO.

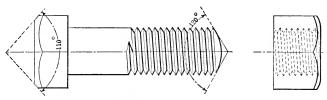
1/2 i	n.	Upset to	₹ in.	Length of	Upset	. 3	in.	Stock 1	eauire	$d, 1\frac{3}{4}i$	in.
5	"	""	3 "	"	- 46		"	"	""		"
3	"	"	\$ in.	"	66	$3\frac{1}{2}$ $3\frac{1}{2}$	"	"	"	T 1	"
125/83/47/8	"	"	T 1 "	"	"	4	"	"	"	$1\frac{1}{2}$ $2\frac{3}{4}$	"
ı°	"	"	11 "	"	"	4	"	"	"	23	"
	"	"	$1\frac{1}{4}$ " $1\frac{3}{8}$ "	"	"	41	"	"	66	$2\frac{1}{4}$	"
τį	"	"		"	"	$4\frac{1}{2}$ $4\frac{1}{2}$	"	"	"		"
T 3	"	44	T 5 "	"	"	5	"	"	"	$2\frac{1}{2}$ $2\frac{1}{8}$	"
ΤÎ	"	"	T 3 "	"	"	5	"	"	"		"
TI	"	"	T 7 "	"	"	5	"	"	"		"
I 4 I 8 I 8 I 2 I 2 I 2 I 2 I 8 I 7 I 8 I 7 I 8	"	"	1½ " 1½ " 1½ " 1½ " 1½ " 1½ " 1½ " 1½ "	"	"	51	"	"	"	$2\frac{7}{8}$ $1\frac{7}{8}$ $1\frac{3}{4}$	"
T 3	"	"	2 "	"	"	$5\frac{1}{2}$ $5\frac{1}{2}$	"	"	44	T 3	"
1 7 6	"	"		"	"	6	"	"	"		"
2	"	"	23 "	"	"	6	"	"	"		"
2	"	"	$2\frac{1}{8}$ " $2\frac{1}{8}$ " $2\frac{1}{2}$ " $2\frac{1}{8}$ " $2\frac{1}{8}$ " $2\frac{7}{8}$ "	"	"	6	"	"	"	2 1 2 1	"
	"	"	25 "	"	"	$6\frac{1}{2}$	"	"	"	$3\frac{1}{2}$ $2\frac{1}{2}$	"
21	"	"	27 "	"	"	$6\frac{1}{2}$	"	"	"	$2\frac{1}{4}$	"
$2\frac{1}{4}$ $2\frac{1}{2}$ $2\frac{3}{4}$	"	"		"	"	7	"	"	"		"
3	"	"	$\frac{3\frac{1}{8}}{3\frac{1}{2}}$ "	"	"	7	"	"	"		"



STOCK REQUIRED TO MAKE MANUFACTURER'S STANDARD BOLT HEADS AND NUTS—ROUGH

NATIONAL MACHINERY CO.

		Bolt I		GON		Nuts Square or Hexagon							
	Short Stock Diameter Requir						ort			Size Stock Required			
olt	Diar	neter	S	for i	Upset	Diai	neter	r Hole	s	W	idth	Thick- ness	
Size of Bolt	Sq.	Hexa- gon	Thickness	Sq.	Hexa- gon	Sq. Hexa-		Diameter Hole	Thickness	Sq.	Hexa- gon	Square or Hexa- gon	
in.	in.	in.	in.	in.	in.	in.	in.	in.	in.	in.	in.	in.	
14 T 38 T 13 O 5 8 8 4 T 5 I 10 14 8 6 12 5 6 8 4 7 5 I 1 I I I I I I I 2 2 14	28 16 12 12 14 15 16 17 17 18 18 18 18 18 18 18 18 18 18 18 18 18	176 772 86 232 166 232 176 66 175 175 175 175 175 175 175 175 175 175	16 1 4 2 2 2 1 4 8 8 6 7 7 1 6 6 2 2 2 1 6 6 1 1 1 6 1 1 1 1 1 1 1	137 118 138 117 1137 1138 115 125 221 213 214 332 216 332 216 332 216 332 216 332 216 332 216 332 216 332 216 332 216 332 216 332 216 216 216 216 216 216 216 216 216 21	23 2 2 3 2 2 3 2 3 2 3 2 3 2 3 2 3 2 3	12 58 84 78 T 1 18 14 12 24 24 2 24 3 3 3 4 4 14 4 14 4 14	12 555 24 75 1 15 14 15 15 55 75 4 15 15 15 15 15 15 15 15 15 15 15 15 15	722 932 112 122 76 16 12 27 16 17 16 17 17 17 17 17 17 17 17 17 17 17 17 17	14 Ki 38 7 I 12 Gi 15 24 7 8 I 15 14 58 12 58 54 7 8 I I I I I I I I I I I I I I I I I I	152 132 232 232 166 167 1176 1176 236 25 25 25 25 25 35 25 25 35 25 25 25 25 25 25 25 25 25 25 25 25 25	13 13 63 13 15 15 15 15 15 15 15 15 15 15 15 15 15	10 1 12 12 12 12 12 12 12 12 12 12 12 12 1	
$2\frac{1}{4}$ $2\frac{1}{2}$	3 8	31'8 318	116 17	5 53	416 418	$4\frac{1}{4}$	3 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4	2 2 1/4	2 ½ 2 ½ 2 ½	4 8 4 8	3 8 4 18	2 5 2 5 8	
23	4 1/8	418	216	532	5 8	4 3 4	41/2	$2\frac{7}{16}$	23	4 5	4 3 8	2 7/8	
3	41/2	41 ⁹ 6	21/4	6 _{1 8}	511	5	43	2 1 1 6	3	4 7 8	4 5	31/8	



STOCK REQUIRED TO MAKE UNITED STATES STANDARD BOLT HEADS AND NUTS-ROUGH

NATIONAL MACHINERY CO.

		OLT HEA			Nuts Square or Hexagon						
Size of Bolt	Short Diam- eter	Thick-ness		Required Jpset	Short Diam- eter	Diameter Hole	Thick -	Req Squa	Stock uired re or agon		
	ctci		Square	Hexa- gon	cter			Width	Thick- ness		
in.	in.	in.	in.	in.	in.	in.	in.	in.	in.		
16 16 38 776 12 96 58 34	13 02 16 52 12 16 17 6 17 6 17 6 17 6 17 6 17 6 17	14 18 18 7 1 30 18 50 88 18 30 0	13 30 12 15 60 70 70 12 1 1 1 2 7 2 12 2 2 2 2 2 2	132 113 114 15 12 15 15 115 115 115 115 115 233 238	10 10 10 10 10 10 10 10 10 10 10 10 10 1	1% scant 1 scant 1 scant 1 scant 1 scant 1 scant 2 scant 3 scant 2 full 5 scant 4 scant 3 scant 3 scant 3 scant 3 scant	14 16 28 28 76 12 9 16 5 8 7 8	·; 17 18 18 18 28 28 28 28 28 18 18	152 132 132 132 132 132 132 132 143 143 143 143 144 145		
1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	$ \begin{array}{c} 1\frac{1}{16} \\ 2 \\ 2\frac{3}{16} \\ 2\frac{3}{8} \end{array} $	1 1 1 3 2 1 1 3 6	3 3 ¹ / ₄ 31 ⁸ / ₈ 31 ³ / ₈	2 ¹⁹ / ₃₂ 2 ²⁷ / ₃₂ 3 31 ⁵ / ₆	$ \begin{array}{c} 1\frac{13}{16} \\ 2 \\ 2\frac{3}{16} \\ 2\frac{3}{8} \end{array} $	15 full 116 full 152 full 132 full	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	$1\frac{3}{4}$ $1\frac{7}{8}$ $2\frac{1}{18}$ $2\frac{1}{4}$	1 1 3 6 1 5 1 5 1 5 5 1 5 5 1 5 5 1 5 5 1 5 5 1 5 5 1 5 5 1 5		
1 5 1 3 1 3 1 7	$2\frac{9}{16}$ $2\frac{3}{4}$ $2\frac{15}{16}$	1 3 5 1 3 5 1 3 5 1 3 5 5 5 5 5 5 5 5 5	$4\frac{1}{6}$ $4\frac{3}{8}$ $4\frac{5}{8}$	$3\frac{1}{2}$ $3\frac{3}{4}$ 4	2 8 2 16 2 3 2 15 2 15	132 scant 133 scant 11 scant 18 scant	1 5 1 5 1 3 1 7 1 7	2 1 6 2 5 2 1 6 2 5 2 1 6	1 8 1 3 1 7 8		
2 2 1 2 1 2 2 2 3 2 4	$3\frac{1}{8}$ $3\frac{1}{2}$ $3\frac{7}{8}$ $4\frac{1}{4}$	1 1 5 1 5 2 1 8	4 ⁷ / ₈ 51 ⁷ / ₆ 51 ⁵ / ₆ 61 ⁷ / ₆	4 ¹ / ₄ 4 ²⁵ / ₂ 5 ³ / ₁₆ 5 ¹ / ₂	3 1/8 3 1/2 3 7/8 4 1/4	133 scant 131 scant 218 218	2 2 1/4 2 1/2 2 3/4	3 3 ³ / ₈ 3 ³ / ₄ 4 ¹ / ₈	$ \begin{array}{c} 2\frac{1}{8} \\ 2\frac{3}{8} \\ 2\frac{5}{8} \\ 2\frac{7}{8} \end{array} $		
3	4 5	2 1 ⁵	7	61 ₆	45	218 218	3	48 41	31/8		

Table of Board Feet in Pieces 1 to 24 Inches Wide Up to 24 Feet Long

THE table herewith shows the board feet contained in planks of a given width and length. This table is of use especially to the patternmaker but may be of value to any one having to make calculations in board measure. 1

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	24,	2 4 0 8 0 11 1 11 1 2 2 2 2 2 2 8 8 8 8 8 8 4 4 4 4 4 4 4 4
	22,	$\begin{array}{cccccccccccccccccccccccccccccccccccc$
	20,	HHHHHH 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0 0
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	,91	$\begin{array}{cccccccccccccccccccccccccccccccccccc$
	15,	
	14,	HINDUMENTAL CONTRACTOR CONTRACTO
	13,	H G W 4 NO 1/30 O H H H H H H H H G G G G G G G G G G G
ength of Board in Feet	12,	H 2 8 4 7 7 0 0 0 1 1 1 1 1 1 1 1 2 2 2 2 2 2 2 2 2
	11,	$\begin{array}{cccccccccccccccccccccccccccccccccccc$
of Bo	10,	1 и 64 м м м м м м м м м м м м м м м м м м
Lengt	٥,	H H H H H H H H H H H H H H H H H H H
	&	$\begin{array}{c} H H H H H H H H H H H H H H H H H H $
	1,	1 1 2 2 2 2 4 4 7 2 7 0 7 7 7 0 0 0 0 1 1 1 1 1 1 1 1 1 1 1
	,9	11 12 12 12 12 12 12 12 12 12 12 12 12 1
	۶,	「 H H G G G G G G G G G G G G G G G G G
	,4	н н н и и и и и и и 4 4 4 гл
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	2,	$ \begin{array}{cccccccccccccccccccccccccccccccccccc$
	Ι,	Line da de la compansión de la compansió
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-		

Width of Board in Inches

Table gives board feet in an inch thick plank. For other thicknesses multiply by thickness in inches.

QUICK WAY OF ESTIMATING LUMBER FOR A PATTERN

MULTIPLY length, breadth, and thickness in inches together and

this by 7, pointing off three places.

Board 8 inches wide, 18 inches long, 1 inch thick. $8 \times 18 \times 1 \times 7 = 1.008$ square feet. This is .008 too much, but near enough for most work. Board $1\frac{1}{2} \times 10 \times 36 = 540 \times 7 = 3.780$. The correct answer is 3.75.

Table Giving Proportionate Weight of Castings to Weight of Wood Patterns

A Pattern Weighing One Pound Made of (Less weight of Core Prints)	Cast Iron	Brass	Copper	Bronze	Bell Metal	Zinc
Pine or Fir. Oak Beech Linden Pear Birch Alder Mahogany Brass	16 9 9.7 13.4 10.2 10.6 12.8 11.7 0.85	18.8 10.1 10.9 15.1 11.5 11.9 14.3 13.2	19.7 10.4 11.4 16.7 11.9 12.3 14.9 13.7 0.99	19.3 10.3 11.3 15.5 11.8 12.2 14.7 13.5 0.98	17 10.9 11.9 16.3 12.4 12.9 15.5 14.2	15.5 8.6 9.1 12.9 9.8 10.2 12.2 11.2 0.81

DEGREES OBTAINED BY OPENING A TWO-FOOT RULE

Degrees	Inches	Degrees	Inches	Degrees	Inches
I 2	.21	1 5 20	3.12 4.17	55 60	11.08
3	.633	25 30	5.21 6.21	65	12.89 13.76
5 7·5	1.04	35 40	7.20 8.21	75 80	14.61 15.43
10	2.09 3.015	45 50	9.20 10.12	85 90	16.21 16.97

Open a two-foot rule until open ends are distance apart given in table when degrees given in table can be scribed. Same results can be had with two 12-inch steel scales placed together at one end.

WEIGHT OF FILLETS

To facilitate the calculations of the weights of the different parts of a machine from the drawings, the accompanying table of areas or volumes of fillets having radii from $\frac{1}{16}$ to 3 inches can be used. It has been calculated for fillets connecting sides that are at right angles to each other.

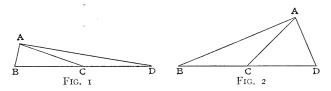
TABLE OF AREAS OR VOLUMES OF FILLETS

Radius of Fillet in Inches	Area or Volume of Fillet in Sq. or Cubic Inches	Radius of Fillet in Inches	Area or Volume of Fillet in Sq. or Cubic Inches
71-80 ° 71-14 ° 70-20 ° 71-12 ° 16 50 ° 1-16 71-17 ° 71 ° 71 ° 71 ° 71 ° 71 ° 71 ° 7	.0008 .0033 .0075 .0134 .0209 .0302 .0410 .0537 .0678 .0838 .1013 .1207 .1417 .1643 .1886 .2146 .2423 .2716 .3026 .3353 .3697 .4057		.5240 .5667 .6119 .6572 .7050 .7543 .8056 .8584 .9129 .9690 1.0269 1.0269 1.0475 1.2105 1.2749 1.3413 1.4086 1.4787 1.5500 1.6229 1.6869 1.7739 1.8518
$1\frac{1}{2}$.4829	3	1.9314

To find the volume of a fillet by this table when the radius and length are given, multiply the value in the table opposite the given radius by the length of the fillet in inches, and this result multiplied by the weight of a cubic inch of the material will give the weight of the fillet.

LAYING OUT A SQUARE CORNER

It sometimes happens that we wish to lay out a perfectly square corner and have no square of any kind handy. Here is a way that requires nothing but a scale or rule, or even a straight stick without any graduations whatever will do. Using this stick, draw a line as $A \ C$ (Fig. 1) and at one end of this draw the line $B \ D$ at any angle. This line must be straight, twice as long as $A \ C$ and



of equal length each side of the point C. Then if you join points

DAB, you have an exact right-angle or square corner.

Fig. 2 is simply another example of this, in which the line $A\ C$ has been drawn at a very different angle to show that it works in any position. Joining the ends $D\ A\ B$ as before also gives an exact right angle.

ANOTHER METHOD

Another method is by what is known as the 6, 8 and 10 rule. This means that if a triangle has sides in the ratio of 6, 8 and 10, the angle is 90 degrees. Lay down a line 6 units long, either inches, feet or yards. Lay off another line 8 units long as nearly right angles as possible. Measure across the ends of the two lines and adjust until this distance is 10 units, which makes it a right angle. These distances may be 3, 4 and 5, 12, 16 and 20 or any combination in this ratio. It is largely used in laying out large corners.

SPEED FOR WOOD TURNING

A good average speed for a wood turning lathe is a surface or cutting speed of from 1,000 to 1,500 ft. per minute. Where work does not exceed 1-in. diameter the lathe may be run 3,000 r.p.m.; for 2-in. stock 2,500; for 3-in., 2,000, or a little less, and for larger stock, the speed is reduced in proportion.

COOLING HOT BEARINGS

A hot box can be cooled by pouring sulphur on the bearing. It melts at 220 degrees and puts a smooth surface on both journals and bearings. It fills oil groove but can be dissolved with benzine if machine cannot be stopped. Either stick or flowers of sulphur will do. Graphite is also good but cannot be used where the color is objectionable as in flour mills or other white stock.

WIRE GAGES AND STOCK WEIGHTS

TWIST DRILL AND STEEL WIRE GAGE SIZES

THE Twist Drill and Steel Wire Gage is used for measuring the sizes of twist drills and steel drill rods. Rod sizes by this gage should not be confused with Stubs' Steel Wire Gage sizes. The difference between the sizes of corresponding numbers in the two gages ranges from about .0005 to .004 inch, the Stubs sizes being the smaller except in the cases of a few numbers where the systems coincide exactly.

TWIST DRILL AND STEEL WIRE GAGE SIZES

No. of Gage	Dia. in Inches	No. of Gage	Dia. in Inches	No. of Gage	Dia. in Inches	No. of Gage	Dia. in Inches
1 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19	.2280 .2210 .2130 .2030 .2055 .2040 .2010 .1990 .1960 .1935 .1910 .1850 .1850 .1800 .1770 .1730 .1695	21 22 23 24 25 26 27 28 29 30 31 32 33 34 35 36 37 38	.1590 .1570 .1540 .1520 .1495 .1470 .1440 .1495 .1360 .1285 .1200 .1160 .1130 .1110 .1065 .1040 .1015	41 42 43 44 45 46 47 48 49 50 51 52 53 54 55 56 57 58 59	.0960 .0935 .0890 .0860 .0820 .0810 .0785 .0760 .0730 .0670 .0635 .0595 .0520 .0465 .0430 .0420	61 62 63 64 65 66 67 68 69 70 71 72 73 74 75 76 77	.0390 .0380 .0370 .0360 .0350 .0320 .0310 .02925 .0280 .0260 .0250 .0240 .0225 .0210 .0200 .0180 .0166
20	.1610	40	.0980	60	.0400	80	.0135

STUBS' GAGES

In using Stubs' Gages, the difference between the Stubs Iron Wire Gage and the Stubs Steel Wire Gage should be kept in mind. The Stubs Iron Wire Gage is the one commonly known as the English Standard Wire, or Birmingham Gage, and designates the Stubs soft wire sizes. The Stubs Steel Wire Gage is used in measuring drawn steel wire or drill rods of Stubs' make and is also used by many American makers of drill rods.

DIFFERENT STANDARDS FOR WIRE GAGES 407

DIMENSIONS IN DECIMAL PARTS OF AN INCH

Number	American or Brown	Birm- ingham or Stubs'	Wash- burn & Moen	Trenton Iron	Stubs' Steel Wire	Impe- rial Wire	U. S. Standard
Gage	& Sharpe	Iron Wire	Mfg. Co.	Co.	wire	Gage	for Plate
000000						.464	.46875
00000				.450		.432	.4375
0000	.46	.454	.3938	.400		.400	.40625
000	.40964	.425	.3625	.360	·	.372	·375
00	.3648	.380	.3310	.330		.348	·34375
0	.32486	.340	.3065	.305		.324	.3125
I	.2893	.300	.2830	.285	.227	.300	.28125
2	.25763	.284	.2625	.265	.219	.276	.265625
3	.22942	.259	.2437	.245	.212	.252	.25
4	.20431	.238	.2253	.225	.207	.232	.234375
4 5 6	.18194	.220	.2070	.205	.204	.212	.21875
6	.16202	.203	.1920	.190	.201	.192	.203125
7 8	.14428	.180	.1770	.175	.199	.176	.1875
	.1 2849	.165	.1620	.160	.197	.160	.171875
9	.11443	.148	.1483	.145	.194	.144	.15625
10	.10189	.134	.1350	.130	.191	.128	.140625
11	.090742	.120	.1205	.1175	.188	.116	.125
12	.080808	.109	.1055	.105	.185	.104	.109375
13	.071961	.095	.0915	.0925	.182	.092	.09375
14	.064084	.083	.0800	.080	.180	.080	.078125
15	.057068	.072	.0720	.070	.178	.072	.0703125
16	.05082	.065	.0625	.061	.175	.064	.0625
17	.045257	.058	.0540	.0525	.172	.056	.05625
18	.040303	.049	.0475	.045	.168	.048	.05
_ 19	.03589	.042	.0410	.oto	.164	.010	.04375
20	.031961	.035	.0348	.035	.161	.036	.0375
21	.028462	.032	.03175	.031	.157	.032	.034375
22	.025347	.028	.0286	.028	.155	.028	.03125
23	.022571	.025	.0258	.025	.153	.024	.028125
24	.0201	.022	.0230	.0225	.151	.022	.025
25	.0179	.020	.0204	.020	.148	.020	.021875
26	.01594	.018	.0181	.018	.146	.018	.01875
27	.014195	.016	.0173	.017	.143	.0164	.0171875
28	.012641	.014	.0162	.016	.139	.0149	.015625
29	.011257	.013	.0150	.015	.134	.0136	.0140625
30	.010025	.012	.0140	.014	.127	.0124	.0125
31	.008928	.010	.0132	.013	.120	.0116	.0109375
32	.00795	.009	.0128	.012	.115	.0108	.01015625
33	.00708	.008	.0118	.011	.112	.0100	.009375
34	.006304	.007	.0104	.010	.110	.0092	.00859375
35	.005614	.005	.0095	.0095	.108	.0084	.0078125
36	.005	.004	.0090	.009	.106	.0076	.00703125
37	.004453			.0085	.103	.0068	.006640625
38	.003965]	.008	.101	.0060	.00625
39	.003531			.0075	.099	.0052	
40	.003144			.007	.097	.0048	

WIRE AND DRILL SIZES ARRANGED CONSECUTIVELY

Dia. of Wire	American or B. & S.	B'ham or Stubs' Wire	Stubs' Steel Wire	Twist Drill and Steel Wire	Dia. of Wire	American or B. & S.	B'ham or Stubs' Wire	Stubs' Steel Wire	Twist Drill or Steel Wire
		Gage 1	Yumber				Gage N	Number	•
.00314 .00353 .00397 .004 .0045 .005 .005 .0063 .007 .007 .008 .008 .009 .010 .0113 .012 .0126 .013 .0135 .014 .0145 .015 .015 .0159 .016 .0179 .018 .020 .0201 .021 .022 .0225 .0203 .0201 .021 .022 .0225 .0225 .0223 .024 .025 .026 .027 .028 .0293 .030 .031 .032 .033 .035 .0359	40 39 38 37 36 35 34 33 32 31 30 29 28 27 26 25 24 22 21	Gage 2 36 36 35 34 33 32 31 30 29 28 27 26 25 24 23 22 21 20	80 79 78 77 76 75 74 71 70 69 68 67 66 65 64	80 79 78 77 76 75 74 73 72 71 70 69 68 67 66 66 65	.041 .042 .043 .045 .0453 .0465 .049 .050 .0508 .052 .055 .0635 .0635 .0635 .0641 .065 .067 .070 .072 .073 .075 .077 .0785 .079 .0808 .081 .082 .083 .085 .089 .0997 .0905 .0905 .0905 .1011 .1015	17 16 15 14 13	19 18 17 16 15 14 14 13	58 57 56 55 54 53 52 51 50 49 48 47 46 43 44 43 41 40 39 38	55 54 53 52 54 49 48 47 44 43 42 41 40 39 38
.036 .037 .038 .039 .040	18		63 62 61 62 59	64 63 62 61 60	.103 .104 .106 .1065 .108			37 36 35	37 36

WIRE AND DRILL SIZES ARRANGED CONSECUTIVELY

Dia. of Wire	American or B. & S.	B'ham or Stubs'	Stubs, Steel Wire	Twist Drilland Steel Wire	Dia. of Wire	American or B. & S.	B'ham or Stubs'	Stubs, Steel Wire	Twist Drilland Steel Wire
.100 .110 .111 .112 .113 .1144 .115 .116 .120 .127 .1285 .134 .136 .139 .1405 .143 .1443 .1443 .1443 .147 .148 .1495 .151 .152 .153 .154 .155 .157 .159 .161	9 8 7	II II IO	34 33 32 31 30 29 28 27 26 25 24 23 22 21 20	35 34 33 32 31 30 29 28 27 26 25 24 23 22 21 20	.203 .204 .2043 .2055 .207 .209 .212 .213 .219 .220 .221 .227 .228 .2294 .234 .238 .242 .246 .250 .257 .257 .2576 .259 .266 .272 .272 .272 .281 .2803 .290 .2905	3	6	5 4 3 2 1 ABCDEF	6 5 4 3 2 I
.162 .164 .165 .166 .168 .1695 .172 .173 .175 .177 .178 .180 .1819 .182 .188 .188 .189 .191 .191 .1935 .194 .190	5	7	19 18 17 16 15 14 13 12 11 10 9 8 7	19 18 17 16 15 14 13 12 11 10 9 8 7	.205 .300 .302 .316 .323 .3249 .339 .340 .348 .358 .368 .368 .368 .377 .386 .386 .397 .404 .4096 .413 .425 .454	0000	000000000000000000000000000000000000000	M NOP QR ST UV WXY Z	

STUBS' STEEL WIRE SIZES AND WEIGHTS

As stated in the explanatory note regarding Stubs' Gages at the bottom of page 406 the Stubs steel wire gage is used for measuring drawn steel wire and drill rods of Stubs' make and is also used by various drill rod makers in America.

Stubs' Steel Wire Sizes, and Weight in Pounds per Linear Foot

Letter and No. of Gage	Dia. in Inches	Weight per Foot	No. of Wire Gage	Dia. in Inches	Weight per Foot	No. of Wire Gage	Dia. in Inches	Weight per Foot
ZYXWVUTSRQPONMLKJIHGFEDCBA 1 2 3 4 5 6 78 9	.413 .404 .397 .386 .377 .368 .358 .348 .339 .322 .295 .295 .295 .296 .281 .277 .272 .266 .261 .257 .250 .246 .242 .238 .234 .227 .219 .219 .219 .219 .219 .219 .219 .219	.456 .437 .422 .399 .380 .362 .335 .324 .397 .295 .280 .267 .244 233 .225 .192 .189 .182 .177 .162 .159 .152 .146 .138 .128 .120 .115 .111 .108 .108	10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26 27 28 29 30 31 32 33 34 35 36 37 38 39 40 41 42 43 44	.191 .188 .185 .182 .180 .178 .177 .168 .161 .157 .155 .153 .151 .148 .146 .143 .139 .134 .127 .110 .108 .106 .103 .101 .108 .101 .108 .109 .109 .109 .109 .109 .109 .109 .109	.098 .095 .092 .089 .087 .085 .079 .075 .075 .066 .064 .063 .061 .059 .057 .055 .048 .039 .035 .031 .030 .028 .028 .027 .026 .026 .026	46 47 48 49 50 51 52 53 54 55 56 67 68 69 70 71 72 73 74 75 76 77 78 79 80	.079 .077 .075 .072 .069 .066 .063 .058 .055 .050 .041 .040 .039 .038 .037 .036 .035 .031 .030 .029 .027 .026 .024 .023 .022 .028 .018 .016	.017 .016 .015 .014 .013 .012 .009 .008 .007 .0045 .0042 .0040 .0039 .0037 .0026 .0024 .0022 .0020 .0018 .0015 .0014 .0013 .0011 .0009 .0007
	-		45	.081	.018			

MUSIC WIRE SIZES

No. of Gage	Wash- burn & Moen	Webster & Horsefall	No. of Gage	Wash- burn & Moen	Webster & Horsefall	No. of Gage	Wash- burn & Moen	Webster & Horsefall
8-0	.0083		6	.0215	.016	19	.0414	.043
7-0	.0087	i l	7 8	.023	.018	20	.0434	.045
6-0	.0095		8	.0243	.020	21	.046	.047
5-0	.010		9	.0256	.022	22	.0483	.052
4-0	.011	.006	10	.027	.024	23	.051	.055
3-0	.012	.007	II	.0284	.026	24	.055	.059
2-0	.0133	.008	12	.0296	.029	25	.0586	.061
1-0	.0144	.009	13	.0314	.031	26	.0626	.065
1	.0156	.010	14	.0326	.033	27	.0658	.070
2	.0166	110.	15	.0345	.035	28	.072	.072
3	.0178	.012	16	.036	.037	29	.076	.077
4	.0188	.013	17	.0377	.039	30	.080	.083
5	.0202	.014	18	.0395	.041	W.		

WEIGHTS OF SHEET STEEL AND IRON UNITED STATES STANDARD GAGE (Adopted by U. S. Government, July 1, 1893)

Number	App. Thickness		er Sq. Foot	No.	App. Thickness	WEIGHT PI	er Sq. Foot
Gage	TillCkiless	Steel	Iron	Gage	TillCkiless	Steel	Iron
0000000	-5	20.320	20.00	17	.05625	2.286	2.25
000000	.46875	19.050	18.75	18	.05	2.032	2.
00000	·4375	17.780	17.50	19	.04375	1.778	1.75
0000	.40625	16.510	16.25	20	.0375	1.524	1.50
000	·375	15.240	15.00	21	.03437	1.397	1.375
00	-34375	13.970	13.75	22	.03125	1.270	1.25
0	.3125	12.700	12.50	23	.02812	1.143	1.125
I	.28125	11.430	11.25	24	.025	1.016	I.
2	.26562	10.795	10.625	25	.02187	.903	.875
3	.25	10.160	10.00	26	.01875	.762	•75
4	.23437	9.525	9.375	27	.01718	.698	.687
5 6	.21875	8.890	8.75	28	.01562	.635	.623
	.20312	8.255	8.125	29	.01406	.571	.562
7 8	.1875	7.620	7.5	30	.0125	.508	-5
	.17187	6.985	6.875	31	.01093	•440	•437
9	.15625	6.350	6.25	32	.01015	.413	.406
10	.14062	5.715	5.625	33	.00937	.381	-375
11	.125	5.080	5.00	34	.00859	.349	•343
I 2	.10937	4.445	4.375	35	.00781	.317	.312
13	.09375	3.810	3.75	36	.00703	.285	.281
14	.07812	3.175	3.125	37	.00664	.271	.265
15	.07031	2.857	2.812	38	.00625	.254	.25
16	.0625	2.540	2.50				

Weight of 1 cubic foot is assumed to be 487.7 lbs. for steel plates and 480 lbs. for iron plates.

WEIGHTS OF STEEL, WROUGHT IRON, BRASS AND COPPER PLATES

AMERICAN OR BROWN & SHARPE GAGE

No.	Thickness	WEI	GHT IN LBS. I	PER SQUARE F	оот
of Gage	in Inches	Steel	Iron	Brass	Copper
of Gage OOOO OO I 2 3 4 5 6 7 8 9 IO II I2 I3 I4 I5 I6 I7 I8 I9 20 21 22 23 24 25	in Inches .46 .4996 .3648 .3249 .2893 .2576 .2294 .2043 .1819 .1620 .1443 .1285 .1144 .1019 .0908 .0808 .0720 .0641 .0571 .0508 .0453 .0403 .0359 .0320 .0285 .0253 .0226 .0201 .0179	Steel 18.77 16.71 14.88 13.26 11.80 10.51 9.39 8.34 7.42 6.61 5.89 5.24 4.67 4.16 3.70 3.30 2.94 2.62 2.33 2.07 1.85 1.64 1.46 1.31 1.16 1.03 .922 .820 .730	Iron 18.40 16.38 14.59 13.00 11.57 10.30 9.18 8.17 7.28 6.48 5.77 5.14 4.58 4.08 3.63 3.23 2.88 2.56 2.28 2.03 1.81 1.61 1.44 1.28 1.14 1.01 .904 .804 .716	Brass 19.688 17.533 15.613 13.904 12.382 11.027 9.819 8.745 7.788 6.935 6.175 5.499 4.898 4.361 3.884 3.458 3.080 2.743 2.442 2.175 1.937 1.725 1.536 1.367 1.218 1.085 .966 .860 .766	Copper 20.838 18.557 16.525 14.716 13.105 11.670 10.392 9.255 8.242 7.340 6.536 5.821 5.183 4.616 4.110 3.660 3.260 2.903 2.585 2.302 2.050 1.825 1.626 1.448 1.289 1.148 1.023 .910 .811
21 22 23 24	.0285 .0253 .0226 .0201	1.16 1.03 .922 .820	1.14 1.01 .904 .804	1.218 1.085 .966 .860	1.289 1.148 1.023 .910
33 34 35 36 37 38 39 40	.0071 .0063 .0056 .0050 .0045 .0040 .0035	.290 .257 .228 .190 .169 .151 .134	.284 .252 .224 .188 .167 .149 .132	.303 .269 .240 .214 .191 .170 .151	.321 .286 .254 .226 .202 .180 .160

WEIGHTS OF STEEL, WROUGHT IRON, BRASS AND COPPER PLATES BIRMINGHAM OR STUBS' GAGE

No. of	Thickness	WEI	GHT IN LBS. F	PER SQUARE F	`00Т
Gage	in Inches	Steel	Iron	Brass	Copper
0000	•454	18.52	18.16	19.431	20.556
000	.425	17.34	17.00	18.190	19.253
00	.380	15.30	15.20	16.264	17.214
0	.340	13.87	13.60	14.552	15.402
1	.300	12.24	12.00	12.840	13.590
2	.284	11.59	11.36	12.155	12.865
3	.259	10.57	10.36	11.085	11.733
4	.238	9.71	9.52	10.186	10.781
5	.220	8.98	8.80	9.416	9.966
6	.203	8.28	8.12	8.689	9.196
7 8	.180	7.34	7.20	7.704	8.154
8	.165	6.73	6.60	7.062	7.475
9	.148	6.04	5.92	6.334	6.704
10	.134	5.47	5.36	5.735	6.070
11	.120	4.90	4.80	5.137	5.436
12	.100	4.45	4.36	4.667	4.938
13	.095	3.88	3.80	4.066	4.303
14	.083	3.39	3.32	3.552	3.769
15	.072	2.94	2.88	3.081	3.262
16	.065	2.65	2.60	2.782	2.945
17	.058	2.37	2.32	2.482	2.627
18	.049	2.00	1.96	2.097	2.220
19	.042	1.71	1.68	1.797	1.902
20	.035	1.43	1.40	1.498	1.585
21	.032	1.31	1.28	1.369	1.450
22	.028	1.14	1.12	1.198	1.270
23	.025	1.02	1.00	1.070	1.132
24	,022	.898	.88	.941	•997
25	.020	.816	.80	.856	.906
26	.018	•734	.72	.770	.815
27	.016	.653	.64	.685	.725
28	.014	.571	.56	•599	.634
29	.013	.530	.52	•556	.589
30	.013	.490	.48	.514	.544
31	.010	.408	.40	.428	•453
32	.000	.367	.36	.385	.408
33	.008	.326	•32	•342	.362
33 34	.007	.286	.28	.2996	.317
	.007	.204	.20	.2990	.227
35 36	.003	.163	.16	.171	.181
3~	.004	.103	•••	''/'	.101
				ĺ	
	i				

WEIGHTS OF STEEL, IRON, BRASS AND COPPER WIRE
AMERICAN OR BROWN & SHARPE GAGE

					
No. of	Dia.	WEIG	HT IN LBS, PE	R 1000 LINEA	R FEET
Gage	Inches	Steel	Iron	Brass	Copper
0000	.4600	566.03	560.74	605.18	640.51
000	.4096	448.88	444.68	479.91	507.95
00	.3648	355.99	352.66	380.67	402.83
0	.3247	282.30	279.67	301.82	319.45
1	.2893	223.89	221.79	239.35	253.34
2	.2576	177.55	175.89	189.82	200.91
3	.2294	140.80	139.48	150.52	159.32
4	.2043	111.66	110.62	119.38	126.35
4 5 6	.1819	88.548	87.720	94.666	100.20
6	.1620	70.221	69.565	75.075	79.462
7 8	.1447	55.685	55.165	59.545	63.013
8	.1285	44.164	43.751	47.219	49.976
9	.1144	35.026	34.699	37.437	39.636
10	.1019	27.772	27.512	29.687	31.426
II	.0907	22.026	21.820	23.549	24.924
12	.0808	17.468	17.304	18.676	19.766
13	.0720	13.851	13.722	14.809	15.674
14	.0641	10.989	10.886	11.746	12.435
15	.0571	8.712	8.631	9.315	9.859
16	.0508	6.909	6.845	7.587	7.819
17	.0453	5.478	5.427	5.857	6.199
18	.0403	4.344	4.304	4.645	4.916
19	.0359	3.445	3.413	3.684	3.899
20	.0320	2.734	2.708	2.920	3.094
21	.0285	2.167	2.147	2.317	2.452
22	.0253	1.719	1.703	1.838	1.945
23	.0226	1.363	1.350	1.457	1.542
24	.0201	1.081	1.071	1.155	1.223
25	.0179	.8571	.8491	.9163	.9699
26	.0159	.6797	.6734	.7267	.7692
27	.0142	.5391	.5340	.5763	.6099
28	.0126	.4275	.4235	.4570	. 483 7
29	.0113	.3389	.3358	.3624	. 383 5
30	.0100	.2688	.2663	.2874	.3042
31	.0089	.2132	.2113	.2280	.2413
32	.0080	.1691	.1675	.1808	.1913
33	.0071	.1341	.1328	.1434	.1517
34	.0063	.1063	.1053	.1137	.1204
35	.0056	.0844	.0836	.0901	.0956
36	.0050	.0668	.0662	.0715	.0757
37 38	.0045	0530	.0525	.0567	.0600
38	.0040	.0420	.0416	.0449	.0475
39	.0035	.0333	.0330	.0356	.0375
40	.0031	.0264	.0262	.0282	.0299

WEIGHTS OF IRON, BRASS, AND COPPER WIRE BIRMINGHAM OR STUBS' GAGE

No.	Dia.	WEIGHT IN	LBS. PER 1000 L	INEAR FEET
of Gage	in Inches	Iron	Brass	Copper
0000 000 000 00 00 1 2 3 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26 27 28 29 30 31 33 34 35 36 36 37 37 37 38 38 38 38 38 38 38 38 38 38 38 38 38	.454 .425 .380 .340 .300 .284 .259 .238 .220 .203 .180 .165 .148 .134 .120 .109 .095 .083 .072 .065 .058 .049 .042 .035 .022 .028 .025 .020 .018 .016 .014 .013 .012 .010 .009 .008 .007 .005 .008	546.21 478.65 382.66 306.34 238.50 213.74 177.77 150.11 128.26 109.20 85.86 72.14 58.05 47.58 38.16 31.49 23.92 18.26 13.73 11.19 8.92 6.36 4.67 3.25 2.71 2.08 1.66 1.28 1.06 .863 .680 .529 .438 .382 .266 .212 .167 .133 .066 .046	589. 29 516.41 412.84 330.50 257.31 230.60 191.79 161.95 138.37 117.82 92.63 77.83 62.62 51.34 41.17 33.97 25.80 19.70 14.82 12.08 9.62 6.86 5.04 3.52 2.93 2.24 1.79 1.39 1.14 .926 .732 .560 .483 .412 .286 .232 .183 .140 .071 .048	623.2 546.1 436.6 349.5 272.1 243.9 202.8 171.3 146.3 124.6 97.96 82.31 66.23 54.29 43.54 35.92 27.29 20.83 15.67 12.77 10.17 7.259 5.333 3.704 3.096 2.370 1.890 1.463 1.209 .979 .774 .592 .511 .435 .302 .2444 .193 .148 .075 .052

WEIGHTS OF STEEL AND IRON BARS PER LINEAR FOOT

		C _n	EEL		Iro	NT.
Dia. or Dis- tance Across			per Foot		Weight p	
Flats	Round	Square	Hexagon	Octagon	Round	Square
1 6	.010	.013	.012	.011	.010	.013
16 1	.010	.053	.046	.044	.041	.052
1 8 3 16	.094	.119	.103	.099	.092	.117
1 6 1	.167	.212	.185	.177	.164	.208
5	.261	.333	.288	.277	.256	.326
. 너ቀ 5.1 ⁶ თვი 7.1 ⁶ - Iza 0.1 ⁶ თვი ¹ 14 თვი 14 თვი	-375	.478	.414	.398	.368	.469
37.	.511	.651	.564	-542	.501	.638
10	.667	.850	.737	.708	.654	.833
18	.845	1.076	.932	.896	.828	1.055
5	1.043	1.328	1.151	1.107	1.023	1.302
116	1.262	1.608	1.393	1.331	1.237	1.576
3	1.502	1.913	1.658	1.584	1.473	1.875
$\frac{13}{16}$	1.763	2.245	1.944	1.860	1.728	2.201
$\frac{7}{8}$	2.044	2.603	2.256	2.156	2.004	2.552
$\frac{\frac{7}{8}}{\frac{15}{16}}$	2.347	2.989	2.591	2.482	2.301	2.930
I	2.670	3.400	2.947	2.817	2.618	3.333
$1\frac{1}{16}$	3.014	3.838	3.327	3.182	2.955	3.763
1 1 6	3.379	4.303	3.730	3.568	3.313	4.219
1 1 3 6	3.766	4.795	4.156	3.977	3.692	4.701
I 1/4	4.173	5.312	4.605	4.407	4.091	5.208
$1\frac{5}{16}$	4.600	5.857	5.077	4.858	4.510	5.742
$1\frac{3}{8}$	5.049	6.428	5.571	5.331	4.950	6.30 2 6.888
$1\frac{7}{16}$	5.518	7.026	6.631	5.827	5.410	7.500
$1\frac{1}{2}$ $1\frac{9}{16}$	6.008	7.650		6.344	5.890	8.138
1 1 6 7 5	6.520	8.301 8.978	7.195 7.776	6.905 7.446	6.392	8.802
$1\frac{5}{8}$ $1\frac{1}{1}\frac{1}{6}$	7.051 7.604	9.682	8.392	8.027	7.455	9.492
$1\frac{1}{1}\frac{1}{6}$ $1\frac{3}{4}$	8.178	10.41	9.025	8.635	8.018	10.21
$1\frac{1}{1}\frac{3}{6}$	8.773	11.17	9.682	9.264	8.601	10.95
1 1 6 1 7	9.388	11.95	10.36	9.204	9.204	11.72
$1\frac{7}{8}$ $1\frac{1}{1}\frac{5}{6}$	10.02	12.76	11.06	10.58	9.828	12.51
2	10.68	13.60	11.79	11.28	10.47	13.33
2 1/8	12.06	15.35	13.31	12.71	11.82	15.05
$2\frac{1}{4}$	13.52	17.22	14.92	14.24	13.25	16.88
$2\frac{3}{8}$	15.07	19.18	16.62	15.88	14.77	18.80
$2\frac{1}{2}$	16.69	21.25	18.42	17.65	16.36	20.83
$2\frac{5}{8}$ $2\frac{3}{4}$	18.40	23.43	20.31	19.45	18.04	22.97
$2\frac{3}{4}$	20.20	25.71	22.29	21.28	19.80	25.21
$2\frac{7}{8}$	22.07	2Š.10	24.36	23.28	21.64	27.55
3	24.03	30.60	26.53	25.36	23.56	30.00
3 1 8	26.08	33.20	28.78	27.50	25.57	32.55
$3\frac{1}{4}$	28.20	35.92	31.10	29.28	27.65	35.21
3 🖁	30.42	38.78	33.57	32.10	29.82	37.97
3½	32.71	41.65	36.10	34.56	32.07	40.83
38	35.09	44.68	38.73	37.05	34.40	43.80
지 3 14 3 8 1 2 5 8 3 4 7 8 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3	37.56	47.82	41.45	39.68	36.82	46.88
	40.10	51.05	44.26	42.35	39.31	50.05
4	42.73	54.40	47.16	45.12	41.89	53.33

WEIGHTS OF BRASS, COPPER AND ALUMINUM BARS PER LINEAR FOOT

Dia. or		Brass			PER		IINUM
Dis- tance	W	eight per F	oot	Weight	per Foot	Weight	per Foot
Across Flats	Round	Square	Hexagon	·Round	Square	Round	Square
16	.011	.014	.013	.012	.015	.003	.001
Į.	.045	.055	.048	.047	.060	.014	.018
18	.100	.125	.108	.106	.135	.032	.041
1/4	.175	.225	.194	.189	.241	.057	.072
15	.275	.350	.301	.296	•377	.089	.114
38	-395	.510	.436	.426	.542	.128	.163
76	.540	.690	.592	.579	•737	.174	.222
$\frac{1}{2}$.710	.905	•773	.757	.964	.227	.290
16	.900	1.15	.978	.958	1.22	.288	.367
<u>5</u>	1.10	1.40	1.24	1.18	1.51	.356	•453
$\frac{11}{6}$	1.35	1.72	1.45	1.43	1.82	.430	.548
34	1.66	2.05	1.73	1.70	2.17	.516	.652
18	1.85	2.40	2.03	2.00	2.54	.601	.766
-60 05 10 144 5 10 200 7 10 122 9 10 150 110 244 250 7 10 110	2.15	2.75	2.36	2.32	2.95	.697	.888
	2.48	3.15	2.71	2.66	3.39	.800	1.02
I	2.85	3.65	3.10	3.03	3.86	.911	1.16
$1\frac{1}{16}$	3.20	4.08	3.49	3.42	4.35	1.03	1.31
$I_{\frac{1}{8}}$	3.57	4.55	3.91	3.81	4.88	1.15	1.47
$1\frac{3}{16}$	3.97	5.08	4.38	4.27	5.44	1.28	1.64 1.81
$1\frac{1}{4}$	4.41 4.86	5.65 6.22	4.82	4.72	6.01	1.42	
$1\frac{5}{16}$ $1\frac{3}{8}$		6.81	5.33 5.76	5.21	6.63	1.57	2.00
$1\frac{8}{176}$	5.35 5.86	7.45	6.38	5.72 6.26	7.24 7.97	1.72 1.88	2.19 2.40
116	6.37	8.13	6.92	6.81	8.67	2.05	2.40
1 9 1 1 6	6.92	8.83	7.54	7.39	9.41	2.22	2.83
1 5 8 1 5 8	7.48	9.55	8.15	7.99	10.18	2.41	3.06
111	8.05	10.27	8.80	8.45	10.73	2.59	3.30
13	8.65	11.00	9.47	9.27	11.80	2.79	3·55
1 1 3 6	9.29	11.82	10.15	9.76	12.43	2.99	3.81
17	9.95	12.68	10.86	10.64	13.55	3.20	4.08
$1\frac{15}{16}$	10.58	13.50	11.68	II.II	14.15	3.41	4.35
2	11.25	14.35	12.36	12.11	15.42	3.64	4.64
2 1/8	12.78	16.27	13.92	13.67	17.42	4.11	5.24
$2\frac{1}{4}$	14.32	18.24	15.72	15.33	19.51	4.61	5.87
$2\frac{3}{8}$	15.96	20.32	17.52	17.08	21.74	5.14	6.54
$2\frac{1}{2}$ $2\frac{5}{8}$	17.68	22.53	19.44	18.92	24.09	5.69 6.27	7.25
28	19.50	24.83	21.24	20.86	26.56		7.99
23/4	21.40	27.25	23.40	22.89	29.05	6.89	8.53
2 7 8	23.39	29.78	25.82	25.02	31.86	7.52	9.58
3,	25.47	32.43	27.84	27.24	34.69	8.20	10.44
31	30.45	38.77	32.76	31.97	40.71	9.62	12.25
$3\frac{1}{2}$ $3\frac{3}{4}$	35.31	44.96	37.80	37.08	47.22	11.16	14.21
	40.07	51.01	43.56	42.11	53.61	12.81	16.31
4	46.12	58.73	49.44	48.43	61.67	14.56	18.56
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WEIGHTS OF FLAT SIZES OF STEEL IN POUNDS PER LINEAR FOOT

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5	.53I	.665	800	.929	.929 I.o6	1.20	1.33	1.46	09.1	1.86	2.13	2.39	2.66	2.02	3.19	3.72	4.25	5.32	6.38
: :	.638	864.	960.	1.12	1.28	1.43	1.59	1.75	16.1	2.23	2.55	2.87	3.20	3.51	3.83	4.46	5.10		7.66
6	.744	.931	1.12	1.30	1.49	1.67	1.86	2.05	2.23	2.60	2.98	3.35	3.72	4.00	4.46	5.21	5.05	7.44	8.92
:	:	1.07	1.28	1.49	1.70	16.1	2.13	2.34	2.55	2.98	3.40	3.83	4.26	4.68	5.10	5.96	6.80		10.20
1.6 · · · · · · · · · · · · · · · · · · ·	:	1.20	1.44	1.67	16.1	2.15	2.39	2.63	2.87	3.35	3.83	4.30	4.78	5.26		6.69	7.65		11.50
:	:	:	09.1	98.1	2.12	2.39	2.66	2.63	3.19	3.72	4.26	4.79	5.32	5.86		7.44	8.52		12.78
	:	:	1.76	2.04	2.34	2.63	2.92	3.22	3.5I	4.09	4.68	5.26	5.84	6.43	7.01	8.18	9.35	9.35 11.70 14.00	14.00
col-4 :	:	:	:	2.23	2.55	2.86	3.19	3.50	3.83	4.46	5.10	5.74	6.40	7.02		8.92	-	10.20 12.80 15.30	15.30
100	:	:	:	2.41	2.76	3.11	3.45	3.80	4.14	4.83	5.53	6.22	16.9	2.60		9.67	11.10	11.10 13.80 16.60	16.60
s0 ;	:	:	:	:	2.98	3.34	3.72	4.09	4.46	5.21	5.96	6.70	7.46	8.19	8.94	10.42 11.92 14.92 17.88	11.92	14.92	17.88
0 0	:	:	:	:	3.19	3.59	3.98	4.38	4.78	5.58	6.38	7.17	7.97	8.77	9.20	11.20 12.80 15.90 19.10	12.80	15.90	01.61
::	:	:	:	:	:	3.82	4.25	4.68	5.10	5.96	6.80	99.4	8.52	9.36	10.20	10.20 11.92 13.60 17.04 20.40	13.60	17.04	20.40
	:	:	:	:	:	:	4.78	5.27	5.74	6.71	7.65	8.61	9.59	10.54	11.48	10.54 11.48 13.41 15.30 19.17	15.30	19.17	22.95
1.4 · · · ·	:	:	:	:	:	:	:	5.85	6.38	7.45	8.50	9.57	10.65	11.71	12.76	11.71 12.76 14.90 17.00 21.30 25.61	17.00	21.30	25.61
12	:	:	:	:	:	:	:	7.02	1.67	8.94	10.20	11.49	12.78	14.04	15.30	14.04 15.30 17.88 20.40 25.56	20.40		30.

NOS. I TO 25 STITBS' GAGE)	(HOME) CONTROL OF THE PROPERTY
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4	.238		:	:	:	:	:	•	:	:	:	1.76	2.10		3.49	4.18	4.82		6.25
Ŋ	.220	:	:	:	:	:	:	:	:	:	:	1.68	1.99		3.26	3.89	4.53		5.80
9	.203	:	:	:	:	:	:	:		1.00	1.29	1.58	1.88		3.04	3.60	4.20	4.80	5.30
7	.180	:	:	:	:	•	:	:	:	06.		I.44	1.71		2.77	3.28	3.78		4.83
∞	.165	:	:	:	:	.40	.52	t9:	.77	.87		1.35	1.59	2.07	2.54	3.02	3.50		4.46
6	.148	:	:	:	:	.39	-49	.61	.71	.82		1.25	1.46	1.89	2.32	2.75	3.17		4.03
01	.134	:	:	:	:	.38	-46	.58	.65	-77		91.1	1.35	1.74	2.12	2.51	2.90		3.67
11	.120	:	:	81.	.26	.36	.43	.53	19.	.70		1.05	1.22	1.57	16.1	2.27	2.16		3.31
12	601.	:	:	1771.	.256	.334	.413	164.	.570	.650		.965	1.12	1.43	1.76	2.07	2.39		3.01
13	.095	:	:	.170	.237	.306	.377	.445	.514	.580		.855	1.00	1.27	1.55	1.82	2.00		2.65
14	.083	:	:	991.	.220	.280	.340	.400	.460	.520		094.	88.	1.12	1.36	1.61	1.84		2.32
15	.072	:	960.	.144	.20I	.251	.303	.355	.400	.461		299.	-77	66.			19.1		2.02
91	.065	.045	260.	.138	981.	.232	6/2.	.326	.372	.420		609.	.70					1.64	1.82
17	.058	.044	.087	.128	691.	.212	.255	.295	.338	.380	.463	.548	÷9:	.80	.97	1.14	1.30	1.48	1.64
18	640.	.043	.078	.113	.150	.183	.220	.255	162.	.325		.467	÷\$:					1.24	1.39
61	.042	.040	020.	101.	.130	191.	.193	.221	.252	.282		404	.46					1.07	61.1
50	.035	.036	.062	980.	.113	.136	.163	.188	.214	.238		.339	.39					.893	00.
21	.032	.034	.057	180.	toI.	.128	.151	.173	961.	612.		.312	.357					.820	.013
2 2	.028	.031	.051	.072	.093	.112	.133	.152	.174	.192		.275	.315					.718	.799
23	.025	.029	.047	990.	.082	.102	611.	.136	.155	.173	.209	.245	.281					.641	.714
24	.022	920.	.042	.058	.074	060.	Lor.	.121	.137	.154	981.	.218	.249					.566	629
25	.020	.024	.039	.052	890.	180.	960.	011;	.126	.140	691.	761.	.226					.516	.573
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For Weights of Seamless Copper Tubing, add 5 per cent, to the weights above.

HORSE-POWER, BELTS AND SHAFTING

Horse-Power

HORSE-POWER is an arbitrary unit of measurement which has been adopted for measuring the work of engines or machines. It is given as 33,000 foot pounds per minute which means 1 pound lifted 33,000 feet per minute or 33,000 pounds lifted 1 foot per minute or 330 pounds lifted 100 feet per minute, or any combination which gives 33,000 foot pounds per minute.

Steam Engine Horse-Power

In a steam engine it means the effective steam pressure per square inch, times the length of piston movement per revolution in feet, times the piston area in square inches, times the number of revolutions per minute, and all divided by 33,000. This is easily remembered by the formula

PLAN 33000

where .

P = mean effective pressure per square inch.

L = length of a double stroke in feet.

A = area of piston in square inches.

N = number of revolutions per minute.

Electrical Power

As compared with electrical units the mechanical horse-power equals 746 watts or nearly $\frac{3}{4}$ of a kilowatt. So that a kilowatt (1000 watts) equals 1:34 horse power.

Gas Engine Horse-Power

The A. L. A. M. rating for gasoline engines, which means the rating adopted by the American Licensed Automobile Manufacturers, is based on the assumption that the piston speed is 1000 feet per minute in all cases. This gives 1500 revolutions per minute for a 4-inch stroke motor, which is about average practice. Since the defeat of the Selden patent the A. L. A. M. has ceased to exist and this is now known as the S. A. E. standard (Society of Automobile Engineers).

S. A. E. (A. L. A. M.) HORSE-POWER RATING

The formula adopted is $\frac{D^2 \times N}{2\cdot 5}$ and based on 1000 feet per minute piston speed. D is the cylinder bore, N the number of cylinders, and 2.5 a constant, based on the average view of the Mechanical Branch as to a fair conservative rating.

Table of Horse-Power for Usual Sizes of Motors, Based on S. A. E. (A. L. A. M.) Formula

В	ORE		Horse	-Power	
Ins.	M/M	ı Cyl.	2 Cyls.	4 Cyls.	6 Cyls.
$ \begin{array}{r} 2\frac{1}{2} \\ 2\frac{5}{8} \\ 2\frac{34}{4} \\ 2\frac{7}{8} \end{array} $	64 68 70 73	$ \begin{array}{r} 2\frac{1}{2} \\ 2\frac{3}{4} \\ 3 \\ 3\frac{5}{16} \end{array} $	5 5 ^{1/2} 6 6 6 ⁵ / ₈	10 11 12 $\frac{1}{10}$ 13 $\frac{1}{4}$	15 16½ 18½ 18½
3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3 3	76 79 83 85	$ \begin{array}{r} 3\frac{3}{5} \\ 3\frac{1}{1}\frac{5}{6} \\ 4\frac{1}{4} \\ 4\frac{1}{1}6 \end{array} $	$ 7\frac{1}{5} 7\frac{1}{1}\frac{3}{6} 8\frac{1}{2} 9\frac{1}{8} $	$14\frac{2}{5}$ $15\frac{5}{8}$ $16\frac{9}{10}$ $18\frac{1}{4}$	$ \begin{array}{r} 21\frac{3}{5} \\ 23\frac{7}{16} \\ 25\frac{2}{5} \\ 27\frac{2}{5} \end{array} $
$3\frac{1}{2}$ $3\frac{5}{8}$ $3\frac{3}{4}$ $3\frac{7}{8}$	89 92 95 99	$ \begin{array}{c c} 4\frac{9}{10} \\ 5\frac{1}{4} \\ 5\frac{5}{8} \\ 6 \end{array} $	$9\frac{4}{5}$ $10\frac{1}{2}$ $11\frac{1}{4}$ 12	$ \begin{array}{r} 19\frac{3}{5} \\ 20\frac{1}{4} \\ 22\frac{1}{2} \\ 24 \end{array} $	$\begin{array}{r} 29\frac{2}{5} \\ 31\frac{3}{5} \\ 33\frac{3}{4} \\ 36\frac{1}{16} \end{array}$
4 189144 4 199 4 199 4 199 4 199 4 199 4 199 4 199 4 199 4 199 199	102 105 108 111	$\begin{array}{c} 6\frac{2}{5} \\ 6\frac{1}{3}\frac{3}{6} \\ 7\frac{1}{4} \\ 7\frac{5}{8} \end{array}$	$ \begin{array}{c} 12\frac{4}{5} \\ 13\frac{5}{6} \\ 14\frac{1}{2} \\ 15\frac{5}{16} \end{array} $	$ \begin{array}{r} 25\frac{3}{5} \\ 27\frac{1}{4} \\ 28\frac{9}{10} \\ 30\frac{5}{8} \end{array} $	38 ² / ₅ 40 ⁹ / ₁₀ 43 ⁴ / ₅
	114 118 121 124	$\begin{array}{c} 8\frac{1}{10}\\ 8\frac{9}{16}\\ 9\\ 9\frac{1}{2} \end{array}$	16 ¹ / ₈ 17 ¹ / ₈ 18	$ 32\frac{2}{5} 34\frac{1}{4} 36\frac{1}{10} 38 $	48 ³ / ₅ 51 ³ / ₅ ·54 ¹ / ₁₀ 57
$ 5 5 \frac{1}{8} 5 \frac{1}{4} 5 \frac{3}{8} $	127 130 133 137	10 10 $\frac{1}{2}$ 11 11 $\frac{9}{16}$	20 21 22 23	40 42 44 10 46	63 66 ¹ / ₅ 69 ¹ / ₁₀
5 1 1 2 1 2 1 2 2 2 2 2 2 2 2 2 2 2 2 2	140 143 146 149 152	$ \begin{array}{c} 12\frac{1}{10} \\ 12\frac{1}{8} \\ 13\frac{1}{4} \\ 13\frac{1}{136} \\ 14\frac{2}{5} \end{array} $	$24\frac{1}{5}$ $25\frac{5}{16}$ $26\frac{1}{2}$ $27\frac{5}{2}$ $27\frac{5}{2}$ $28\frac{1}{5}$	48 \frac{25}{5} \frac{5}{8} \frac{5}{8} \frac{5}{8} \frac{5}{4} \frac{1}{4} \frac{5}{4} \frac{7}{8} \frac{7}{8} \frac{7}{8} \frac{7}{8} \frac{7}{8} \frac{7}{8} \frac{7}{8} \frac{7}{8} \frac{7}{8} \frac{1}{8} \f	$72\frac{3}{5} \\ 75\frac{1}{16} \\ 79\frac{1}{2} \\ 82\frac{1}{10} \\ 86\frac{5}{5}$

To simplify reading of the above, the horse-power figures are approximate, but correct within one-sixteenth.

DRIVING POWER OF LEATHER BELTS

The question of the proper size of a leather belt for a given power transmission resolves itself into a question of selecting various factors. These factors have been worked out by experiments, by ana-

lytical methods, and in practice.

The horse-power that a belt will transmit depends upon the effective tension and the belt speed. The effective tension depends upon the difference in the tensions of the two sides of the belt and on the surface friction, which depends upon the ratio of the tensions and the angle of belt contact with pulley.

Experiments and practice have shown that a belt of single thickness will stand a stress of 60 pounds per inch of width and give good results, that is, it will only require an occasional taking up and will have a fairly long life. The corresponding values for double and triple belts are 105 and 150 pounds per inch of width provided the

pulleys are not too small.

Experiments have shown that on small pulleys the ratio of the tensions should not exceed 2, on medium pulleys 2.5, and on large pulleys 3. The larger the pulley, the better the contact; also the thinner the belt, the better the contact for the same size of pulley. When the pulley diameter in feet is three times the thickness of the belt in *inches*, or in this proportion, we get equivalent results for different thicknesses of belts. This gives us a method of classifying our pulleys. The belt has to adjust itself in passing over a pulley due to its own thickness. Some adjustment is also necessary on account of the crowning of the pulley. These adjustments account for the different ratios for the various pulley diameters. The effects of the crown and pulley diameters are not usually considered in belt rules, although they should be. The ratios given are for 180 degrees wrap and decrease with less contact.

The creep of the belt depends upon its elasticity and the load, and experiments have shown that this should not exceed I per cent. in good practice. In order to keep this creep below I per cent. it is necessary to limit the difference of tension per inch of width of single belt to 40 pounds. The corresponding values for double and triple belts are 70 and 100 pounds per inch of width. These figures are based on an average value of 20,000 for the running modulus of

elasticity of leather belting.

Table I has been prepared on the basis of these limitations and gives a value for the factor or constant F in the equation

$$H.P. = \frac{V \times W}{F}$$
 or $W = \frac{H.P. \times F}{V}$

in which H P is the horse-power, V the belt velocity in feet per minute,

and W the width in inches.

Table 2 gives corrected values for F, when the arc of contact or wrap is greater or less than 180 degrees. On large pulleys the creep may exceed 1 per cent. if the wrap is over 180 degrees, as the increased friction gives a greater difference of tensions.

To illustrate the use of the tables, we will take the following

examples:

Table I.—Factors for 180 Degrees of Belt Contact

Diameter of pulley	$^{\rm Under}_{8''}$	8"-36"	Over 3 feet	Under 14"	14"-60"	Over 5 feet	Under 21"	21"-84"	Over 7 feet
Thickness of belt	Single	Single	Single	Double I	Double	Double Double	Triple	Triple	Triple
Factor Difference of tensions. Per cent. of creep Ratio of tensions. Tension on tight side	30 0.74 2.00 60	920 36 0.89 2.50 60	830 40 0.99 3.00 60	630 52.5 0.74 2.00 105	520 63 0.89 2.50 105	470 70 0.99 3.00 105	440 75 0.74 2.00 150	370 90 90.89 2.50	330 100 0.09 3.00 150

Table II.—Factors for Varying Degrees of Belt Contact

140° 130° 120°	1330	1170	1030	260	099	500 590 630	540	470	420
150°	1220	1040	930	700	009	530	200	430	380
160°	1180	066	890	029	570	510	480	410	360
170°	1140	950	860	650	540	490	460	390	340
180°	0011	920	830	630	520	470	440	370	330
1900	10701	890	800	019	500	450	420	350	320
200°	1040	860	770	590	480	440	410	340	310
210°	0101	830	750	570	470	430	400	330	300
220°	980	810	730	260	460	420	390	320	290

How much horse-power will a 4-inch single belt transmit at a speed of 4600 feet per minute passing over a 12-inch pulley? The factor is 920, therefore,

$$\frac{4600 \times 4}{920} = 20 \text{ H.P.}$$

How wide should a belt be in order to transmit 50 horse-power at 2000 feet per minute on a 36-inch pulley?

$$W = \frac{50 \times 830}{2000} = 20.7$$
-inch single belt.

This gives a width of single belt beyond the usual limit, 8 inches being considered good practice for the maximum width of a single belt.

$$W = \frac{.50 \times 5^{20}}{2000} = 13 \text{-inch double belt.}$$

How wide should a single belt be in order to transmit 2 horsepower at 600 feet per minute over a 4-in. pulley with 140 deg. wrap? In this case we take the factor 1100 from Table 1 and in Table 2

find a corrected value for 1100 under 140 degrees of 1270.

$$W = \frac{2 \times 1270}{600} = 4.23 \text{-inch single belt.}$$

How wide a belt is required for 300 horse-power at 2000 feet per minute over a 10-foot pulley?

$$W = \frac{300 \times 470}{2000} = 70.5$$
-inch double belt.

This is too wide. Good practice calls for a change to triple at 48 inches unless for some special reason a narrower belt is necessary.

$$W = \frac{300 \times 330}{2000} = 49.5 \text{-inch triple belt.}$$

The results given by these factors are well within working values and the belts will probably transmit 50 per cent. more power than these factors, but at the expense of the life of the belt. A liberal allowance at the beginning means less annoyance, fewer delays in taking up the belts, longer life and less cost for renewals and repairs. Belt speeds of 4000 to 4800 ft. should rarely be exceeded.

Transmission of power in mill work is by gearing, by shafting, by electric motors, and by leather and rope belting, and which of these should be used in a particular case is a problem for the engineer

in charge to determine.

For successful work the pulleys must be large in diameter and must have a smooth surface where the rope bears upon them. The speed and the load on the rope must also be such as experience has shown to be economical. When these conditions are fulfilled, a rope drive is a very satisfactory method of transmitting power.

The table shows the horse-power of driving ropes and the diameter of pulleys that should be used for this purpose. This table takes into consideration the effects of the centrifugal force, so that the strain on the rope is constant on the driving side in transmitting the tabular power, no matter what the speed may be. While many engineers recommend a much larger horse-power, we believe the estimates here given are advisable except in temporary installations.

HORSE-POWER OF MANILA ROPE

Dia.				Ve	locity,	Feet pe	r Minut	e			
of Rope	1,000	1,500	2,000	2,500	3,000	3,500	4,000	4,500	5,000	5,500	6,000
3 4 7 8	2.3	3·3 4·5	4·3 5·9	5.2 7.0	6.0 8.2	6.6	7.2 9.6	7·3 9.8	7.4 10.0	7·3 9.6	6.9
1	4.0	5.9	7.7	9.2	10.6	11.8	12.7	12.9	13.0	12.7	12.0
I 1/8	5.0	7.5	9.7	11.6	13.5	14.9	16.0	16.3	16.7	16.5	15.3
I 1 1 4 3 1 8 1 1 2 5 1 8 3 4 4 1 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4	6.3	9.1	12.0	14.3	16.7	18.5	20.0	20.2	20.7	20.1	18.9
I 3/8	7.5	10.8	14.4	17.4	20.0	22.1	23.7	24.5	24.6	24.0	22.3
$r^{\frac{1}{2}}$	9.0	13.5	17.4	20.7	23.0	26.3	28.7	29.0	29.5	28.6	26.7
I 5/8	10.5	15.5	20.1	24.3	27.9	30.8	32.9	34.1	34.3	33.3	31.0
$1\frac{3}{4}$	12.3	18.0	23.6	28.2	32.7	36.4	38.5	39.4	40.5	38.7	36.0
2	16.0	23.2	30.6	46.8	42.5	46.7	50.0	51.7	52.8	50.6	47.3
$2\frac{1}{4}$	20.0	29.6	38.6	46.6	53.6	59.2	63.6	65.8	66.3	64.4	60.3
$2\frac{1}{2}$	25.0	36.6	47.7	57.5	66.0	71.2	78.0	80.0	81.0	79.0	73.8

DATA OF MANILA TRANSMISSION ROPE

Sope	meter	per ft.	ngth	Allowable	Lengt	h of S Feet	plice,	am. of Inches	Number utions
Diameter of Rope	Square of Diameter	Approx. Wgt. per	Breaking Strength	Maximum All Tension	3 Strands	4 Strands	6 Strands	Smallest Diam. Sheaves, Inch	Maximum Numbof Revolutions
3 4 7 8	.5625	.20	3,950	112	6	8		28	760 650
	.7656	.26	5,400	153	6	8		32	650
1	I.	-34	7,000	200	7	10	14	36	570
I 1/8	1.2656	.43	8,900	253	7	10	16	40	510
$I_{\frac{1}{4}}$	1.5625	.53	10,900	312	7 8	10	16	46	460
1 3 E	1.8906	·53 .65	13,200	378		12	16	50	415
$I^{\frac{1}{2}}$	2.25	.77	15,700	450	8 8	12	18	54	380
1 5/8	2.6406	.90	18,500	528	8	12	18	60	344
I 1814388 112588314	3.0625	1.04	21,400	612	8	12	18	64	330
2	4.	1.36	28,000	800	9	14	20	72	290
$2\frac{1}{4}$	5.0625	1.73	35,400	1,012	9	14	20	82	255
$2 \frac{1}{2}$ $2\frac{1}{4}$ $2\frac{1}{2}$	6.25	2.13	43,700	1,250	10	16	22	90	230

weight of transmission rope	=	.34 × diameter.
Breaking strength		
Maximum allowable tension		
Diameter smallest practicable sheave	=	$36 \times \text{diameter.}$
Velocity of rope (assumed)	=	5,400 feet per minute.

BELT FASTENINGS

The best fastening for a belt is the cement splice. It is far beyond any form of lacing, belt hooks, riveting, or any other method of joining together the ends of a belt. The cement joint is easily applied to leather and to rubber belts, but to make a good cement splice in a canvas belt requires more time and apparatus than is usually at hand. Good glue makes a fine cement for leather belts, and fish glue is less affected by moisture than the other. Many of the liquid glues are fish glue treated with acid so as not to gelatinize when cold. A little bichromate of potash added to ordinary hot glue just before it is used will render it insoluble in water. Both lap and wedge joints are used.

Belt Hooks

There are many styles of belt hooks in use, some of the more common kind being shown in Figs. 1, 2, 3 and 4. Fig. 2 is practically a double rivet, Fig. 3 a malleable iron fastening, although similar hooks have been made of pressed steel, and Fig. 4 is the Blake stud, which has the advantage of not weakening the belt but makes a hump on the outside where the ends turn up. Fig. 5 is the Bristol hook of stamped steel which is driven in the points turned over on the other side. Fig. 6 is the Jackson belt lacing and is applied by a hand machine which screws a spiral wire across the ends of a belt. These are then flattened and a rawhide pin or a heavy soft cord used as a hinge joint between them. These joints are probably equal to 90 per cent. of the belt strength.

Lacing Belts

Belts fastened by lacing are weakened according to the amount of material punched out in making the holes to receive the lacing. It is preferable to lace with a small lacing put many times, through small holes. Such a joint is stronger than a few pieces of wide lacing through a number of large holes. Figs. 7 and 9 illustrate two forms of belt lacing, the latter being far preferable to the other. The lacing shown by Fig. 7 is in a double leather belt 5 inches wide. The width makes no difference as the strength is figured in percentage of the total width. There are four holes in this piece of belt, each hole $\frac{3}{8}$ inch in diameter. The aggregate width thus cut out of the belt is $4 \times \frac{3}{8}$ inch $= \frac{1}{8} = 1\frac{1}{2}$ inches. Then $1.5 \div 5 = 0.30$, or 30 per cent. of the belt has been cut away—nearly one-third of the total strength.

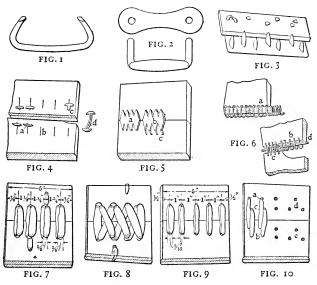
Another Method of Lacing

In Fig. 9 a different method is followed. Instead of there being a few large holes, there are more smaller ones—one fourth more, in fact. There are five holes, each $\frac{3}{15}$ inch in diameter, making a total of $\frac{15}{16}$ inch or 0.9375 ÷ $5=18\frac{3}{4}$ per cent., leaving $81\frac{1}{4}$ per cent. of the total belt strength against 70 per cent. in the belt with large holes. A first-class double leather belt will tear in two under

a strain of about 500 pounds to each lace hole, the strain being

applied in the holes by means of lacings.

The belt shown by Fig. 9 has $81\frac{1}{4}$ per cent. of 1.875 square inches of section, = 1.525 square inches left after cutting out the five holes. This amount is good for $3000 \times 1.875 = 5625$ pounds breaking strain, and as the lacing will tear out under 2500 pounds, it will be seen that we cannot afford to use lacings if the full power of the leather is to be utilized. This, under a factor of safety of 5, would be 1125 pounds to the square inch, or 1125 \times 1.525 = 1715 pounds



Belt Hooks and Lacings

working strain for the belt, or $1715 \div 5 = 343.5$ pounds to each lace. This, too, is too much, as it is less than a factor of safety of 2.

The belt to carry 40 pounds working tension to the inch of width must also carry about 40 pounds standing tension, making a strain of 80 pounds to the inch, or $80 \times 5 = 400$ pounds. This is a better showing, and gives a factor of safety of $2500 \div 400 = 6\frac{1}{4}$. Still, we are wasting a belt of 5625 pounds ultimate strength in order to get from it 400 pounds working strain. This means a factor of safety of over 14 in the body of the belt but of only $6\frac{1}{4}$ at the lacing, which shows the advantage of a cement splice.

Fig. 10 shows a method sometimes used to relieve the lace-holes of some of the strain. Double rows of holes are punched as at a b, and the lacing distributed among them. As far as helping the

strength of the belt is concerned, this does nothing, for all the stress put upon the belt by the lacing at c must be carried by the belt section at a; therefore this way of punching holes does not increase the section strength. Neither does staggering the holes as shown at d and e. The form of hole-punching shown at a b c is desirable for another reason. It distributes the lacing very nicely and does not make such a lump to thump when it passes over the pulleys.

ALINING SHAFTING BY A STEEL WIRE

A STEEL wire is often used for alining shafting by stretching it parallel with the direction of the shaft and measuring from the shaft to the wire in a horizontal direction. This steel wire can also be used for leveling or alining in a direction at right angles to the other,

SAGS OF A STEEL ALINING WIRE FOR SHAFTING

]	Distan	ce in	Feet	, fro	m R	eel to	o Poi	nt o	f Me	asure	men	t		
		10	20	30	40	50	60	70	80	90	100	110	120	130	140
Distance in Feet, from Reel to Point of Support	280 270 260 250 240 230 220 210 200 190 180 170 160 130 120 110 100 90 80 70 60 50 40 30	14 5 4 5 4 5 7 2 7 3 3 3 4 3 6 1 4 4 4 4 5 2 5 2 2 2 4 4 5 4 6 4 5 6 8 8 2 2 2 2 2 5 4 6 5 6 5 6 6 6 6 6 6 6 6 6 6 6 6 6 6	$\frac{1}{2} + \frac{1}{2} + \frac{1}$	1(0.01)+1(+0)+1(1-10)(0.01)0 (0.01) (0.01)+1(0.01)+1(0.01)(0.01)+1(0.01)	7-14-014-05-0-0-0-10-17-0-0-17-0-0-0-0-0-0-0-0-0-0-	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	$\begin{array}{ c c c c c c c c c c c c c c c c c c c$	1	1 1 1 1 1 1 1 1 1 1	3 6 1 2 7 7 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	1 x 1 1 1 1 1 1 1 1 2 2 2 2 2 2 2 2 2 2	1 44646127 16 1684 + 1 4 622365464894 23 32 16834 33 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	1 460 0 1 1 16 3 16 3 3 23 4 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	

by making vertical measurements, if it is stretched under established conditions and if the sags at the points of measurement are known. The accompanying table gives the sags in inches from a truly level line passing through the points of support of the wire, at successive points beginning 10 feet from the reel and spaced 10 feet apart for a No. 17 Birmingham gage high grade piano wire, stretched with a weight of 60 pounds, wound on a reel of a minimum diameter of three and one-half inches and for total distances between the reel and point of support of the wire varying by increments of 10 feet from 40 to 280 feet. Thus a wire of any convenient length, of the kind indicated, can be selected, so long as this length is a multiple of 10 feet and between the limits specified, and the table gives the sags from a truly level line at points 10 feet apart for its entire length when it is stretched under the conditions designated. These sags

SAGS OF A STEEL ALINING WIRE FOR SHAFTING

	Distance in	Feet, fro	om Reel	to Poi	nt of	Mea	suren	nent			
Support	150 1	6c 170	180 19	200	210	220	230	240	250	260	270
Distance in Feet, from Reel to Point of Seet of Point of Seet	$\begin{bmatrix} 1\frac{1}{3}\frac{3}{2} & 1 \\ 1\frac{1}{6}\frac{5}{4} & 1 \end{bmatrix}$	1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	$\begin{array}{c ccccccccccccccccccccccccccccccccccc$		1/21:5/24-7/4 3/8 3/6 3/6 1	I 3/2 I 643/69/44 1 1 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2 2	I 1 5544 147 4 657 3 5 2 6 7 5	574334914 62867 32	11 116 132 164 re in	31 615 64	thes

being known, direct measurements can be made to level or aline a shaft by vertical measurements.

The method was originally developed for alining the propeller shafts of vessels, but it is equally serviceable for semi-flexible shafting, as factory line shafts.

SPEED OF SHAFTING

Line shaft speed varies with machinery it drives. Probably 250 r.p.m. is an average today, with cases of 400 r.p.m. even on 4-in. shafts and 600 to 700 r.p.m. on 2-in. shafts for high speed machinery.

HORSE POWER OF STEEL SHAFTING

FOR LINE SHAFT SERVICE

Sizes	Revolutions per Minute													
Shaft S Inches	100	125	150	175	200	225	250	300	350	400	Max. Dis Between Bearings			
11 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	2.4 4.3 6.7 10.0 14.3 19.5 26.0 33.8 43.0 53.6 65.9 80.0 113.9 156.3 207.9 270.0	3.1 5.3 8.4 12.5 17.8 24.4 32.5 42.2 53.6 67.0 82.4 100.0 142.4 195.3 260.0 337.5	3.7 6.4 10.1 15.0 21.4 29.3 39.0 50.6 64.4 79.4 97.9 120.0 170.8 234.4 311.9 405.0	4-3 7-4 11.7 17.5 24.9 3.1.1 43.5 59.1 75.1 93.8 115.4 140.0 199.3 273.4 363.9 472.5	4.9 8.5 13.4 20.0 28.5 39.0 52.0 67.5 85.8 107.2 121.8 160.0 227.8 312.5 415.9 5,0.0	5.5 9.5 15.11 22.5 32.1 44.1 58.5 75.9 96.6 120.1 148.3 180.0 256.2 351.5 459.9 607.5	6.1 10.5 16.7 25.0 35.6 48.7 65.0 84.4 107.3 134.0 164.8 200.0 284.7 390.6 520.0	7.3 12.7 20.1 30.0 42.7 58.5 78.0 101.3 128.7 158.8 195.7 240.0 341.7 623.9 810.0	8.5 14.8 23.4 49.8 68.2 87.0 118.2 150.3 187.6 230.7 280.0 398.6 546.8 727.9 945.0	16.9 26.8 40.0 57.0 78.0 104.0 135.0 171.6 214.4 243.6 320.0	6.8" 7.2" 8.2" 8.9" 9.6" 10.2" 11.4" 12.5"			
6½ 7 8	343.3 428.8 640.0	429.0 535.9 300.0	514.9 643.1	600.7 750.3 1126.0	686.5 847.5 1280.0	772.4 964.7 1440.0	858.0 1071.0 1600.9	1029.0 1286.0 1920.0	1201.0 1500.0 2240.0	1372.0 1695.0 2560.0				

This table is based on the formula

$$M = \frac{D^3 \times R}{80}$$
. For heavier work use $M = \frac{D^3 \times R}{100}$

For head and jack shafts, supported by bearings close to main sheave or pulley so as to prevent transverse strain, the following formula may be used with safety:

$$M = \frac{D^3 \times R}{125}$$

D = Diameter of shaft in inches.

R =Number of revolutions per minute.

M = Horse-power.

Deflection of shafting from weight of pulleys and draw of belting should not be allowed in excess of .oor per foot, as this action adds very rapidly to the power cost. If this deflection is at a clutch, sleeve or roller bearing, any of them may be ruined easily and quickly. It can be reduced by using more hangers.

SPEEDS OF PULLEYS AND GEARS

THE fact that the circumference of a pulley or gear is always 3.1416 or 3½ times the diameter makes it easy to figure speeds by considering only the diameter of both driver and driven pulleys. Belting from one 6-inch pulley to another gives the same speed to both; but if the driving pulley is 16 inches and the driven pulley only four inches it is clear that the small pulley will turn 4 times for

every turn of the large pulley. If this is reversed and the small pulley is the driver, the large pulley will only make one turn for every four of the small pulley. The same rule applies to gears if the pitch diameter and not the outside diameter is taken. The following rules have been arranged for convenience in finding any desired information about pulley or gear speeds.

HAVING	TO FIND	RULE
Diameter of Driving Pulley Diameter of Driven Pulley Speed of Driving Pulley	Speed of Driven Pulley	Multiply Diameter of Driving Pulley by its Speed and divide by Diameter of Driven Pulley.
Diameter of Driving Pulley Speed of Driving Pulley Speed of Driven Pulley	Diameter of Driven Pulley	Multiply Diameter of Driving Pulley by its Speed and Divide by Speed of Driven Pulley.
Diameter of Driving Pulley Diameter of Driven Pulley Speed of Driven Pulley	Speed of Driving Pulley	Multiply Diameter of Driven Pulley by its Speed and Divide by Diameter of Driving Pulley.
Diameter of Driven Pulley Speed of Driven Pulley Speed of Driving Pulley	Diameter of Driving Pulley	Multiply Diameter of Driven Pulley by its Speed and Divide by Speed of Driving Pulley.

These rules apply equally well to a number of pulley belts together or to a train of gears if *all* the driving and *all* the driven pulley diameters and speeds are grouped together.

TABLES OF CIRCUMFERENTIAL SPEEDS

THE tables on pages 432-435 which give circumferential speeds, can be used for obtaining gear and belt speeds and the speed of

revolving parts of high-speed motors.

For diameters greater than those given in the tables, the speeds can be obtained by adding together the speeds for two diameters whose sum equals that of the diameter for which we require the speed. For example, to find the speed at a 120-inch diameter and 200 revolutions per minute, the following calculation is readily made:

```
Speed for 100-inch diameter — 5236 feet.
Speed for 20-inch diameter — 1047 feet.
Speed for 120-inch diameter — 6283 feet.
```

To interpolate, we can use the values given for speed for 1- to 10-inch diameters, dividing them by 10, 100, 1000, etc., to obtain speeds for tenths, hundredths, thousandths, etc. For instance, if the speed for 550 revolutions per minute and 46.186-inch diameter is required, we proceed as follows:

CIRCUMFERENTIAL SPEEDS IN FEET PER MINUTE (See page 431)

						Revo	lutions	per M	inute			
		50	100	150	200	250	300	350	400	450	500	550
Diameter in Inches	1 2 3 4 5 6 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26 27 28 29 30 31 32	133 26 39 52 105 118 1181 1144 1157 1196 209 2236 2249 2236 249 2236 301 314 314 327 349 357 349 349 369 369 37 38 39 39 39 39 39 39 39 39 39 39 39 39 39	26 52 79 105 1131 1157 236 262 2288 314 349 445 471 471 471 471 471 471 471 471 471 471	39 79 1188 157 1157 1275 3144 471 550 589 668 775 864 982 1021 11060 1130 11178 11178 11178	200 52 105 105 105 209 262 314 367 628 838 890 890 1301 1257 1100 1257 1100 1257 1100 1257 1100 1257 1100 1257 1100 1257 1100 1257 1100 11		I			450 118 236 353 471 589 707 825 942 1,060 1,178 1,296 1,414 1,532 2,003 2,121 2,238 2,356 2,474 2,592 2,710 2,827 2,945 3,063 3,181 3,299 3,417 3,532 3,770	131 262 393 523 524 785 7916 1,077 1,178 1,309 1,440 1,571 1,701 1,832 1,963 2,255 2,356 2,487 2,258 2,356 2,487 3,011 3,142 3,273 3,403 3,273 3,403 3,734 3,734 4,658 4,189	550 144 288 432 576 720 863 1,008 1,152 1,296 1,440 1,728 1,872 2,016 2,160 2,160 2,304 2,442 2,442 2,431 3,168 3,312 3,456 3,600 3,744 3,888 4,032 4,176 4,320 4,464 4,462
	33 34 35 36 37 38 39 40 41 42 43 44 45 46 47 48 49 50	432 445 458 471 484 497 511 524 537 550 563 576 602 615 628 641 655	864 890 916 943 969 995 1021 1047 1073 1100 1125 1178 1204 1231 1257 1283 1309	1296 1335 1375 1414 1453 1492 1532 1571 1610 1649 1728 177 1806 1846 1885	1728 1780 1833 1885 1937 1990 2042 2094 2147 2199 2251 2304 245 2461 2513 2566 2618	2160 2225 2291 2356 2422 2487 2553 2618 2683 2749 2814 2880 2945 3011 3076 3142 3207	2592 2670 2749 2827 2906 3063 3142 3220 3299 3377 3456 3534 3613 3692 3770 3849	3024 3115 3206 3299 3390 3482 3573 3665 3757 3848 3940 4032 4123 4215 4307	3,456 3,565 3,760 3,770 3,875 3,979 4,084 4,189 4,294 4,398 4,503 4,608 4,712 4,817 4,922 5,027 5,131 5,236	3,888 4,006 4,123 4,241 4,359 4,477 4,595 4,712 4,831 4,948 5,066 5,184 5,301 5,419 5,537 5,655 5,773 5,891	4,320 4,451 4,781 4,712 4,843 4,974 5,105 5,236 5,367 5,498 5,629 5,760 5,891 6,152 6,283 6,414 6,545	4,752 4,896 5,040 5,184 5,328 5,472 5,616 5,760 5,904 6,048 6,192 6,336 6,480 6,623 6,768 6,912 7,056 7,200

CIRCUMFERENTIAL SPEEDS IN FEET PER MINUTE (See page 431)

_												
						Reve	olution	s per l	Minute			
				<u> </u>								
		50	100	150	200	250	300	350	400	450	500	550
	51	668	1335	2003	2670	3338	4006	4673	5,341	6,008	6,676	7,343
	52	681	1361	2042	2723	3403	4084	4764	5,445	6,126	6,807	7,487
	53	694	1388	2081	2775	3469	4163	4856	5,550	6,244	6,938	7,631
	54	707	1414	2121	2827 2880	3534	4241	4948	5,655	6,362 6,480	7,069	7,775
	55	720	1440 1466	2100	2032	3600 3665	4320 4398	5040	5,760 5,864	6,597	7,199	7,919 8,063
	56	733	1400	2238	2985	3731	4390	5131	5,969	6,715	7,330 7,461	8,207
	57 58	759	1518	2278	3037	3796	4555	5314	6,074	6,833	7,592	8,351
	59	772	1545	2317	3089	3862	4634	5406	6,178	6,951	7,723	8,495
	60	785	1571	2356	3142	3927	4712	5498	6,283	7,069	7,854	8,639
	61	799	1597	2395	3194	3992	47QI	5589	6,388	7,186	7,985	8,783
	62	799 812	1623	2435	3246	4058	4870	5681	6,493	7,304	7,985 8,116	8,927
	63	825	1649	2474	3299	4123	4948	5773	6,597	7,422	8,247	9,071
	64	838	1676	2513	3351	4189	5027	5864	6,702	7,540	8,378	9,215
	0.5	851	1702	2552	3403	4254	5105	5956	6,807	7,658	8,508	9,359
	66	864	1728	2592	3456	4320	5184	6048	6,912	7,775	8,640	9,503
	67	877	1754	2631	3508	4385	5262	6139	7,016	7,893	8,770	9,647
	68	890	1780	2670	3560	4451	5341	6231	7,121	8,011	8,901	9,791
	69	903	1806 1833	2710 2749	3613 3665	4516	5419 5498	6414	7,226	8,129 8,247	9, 0 32 9,163	9,935
	, 3 71	929	1859	2788	3718	4647	5576	6506	7,435	8,365	9,29.1	10,223
v.	72	943	1885	2827	3770	4712	5655	6597	7,540	8,482	9,425	10,367
Diameter in Inches	73	956	1911	2867	3822	4778	5733	6680	7,644	8,600	9,556	10,511
2	74	969	1937	2006	3875	4843	5812	6781	7,749	8,718	9,687	10,655
_	75	982	1064	2945	3927	4909	5890	6872	7,854	8,836	9,818	10,796
·=	76	995	1990	2985	3979	4974	5969	6964	7,959	8,954	9,948	10,943
Ę	77 78	1008	2016	3024	4032	5040	6048	7056	8,063	9,072	10.079	11,087
ne	78	1021	2042	3063	4084	5105	6126	7147	8,168	9,189	10,210	11,231
iar	79 80	1034	2068	3102	4136	5171	6205	7239	8,273	9,307	10,341	11,375
А	80	1047	2094	3142	4189	5236	6283	7330	8,378	9,425	10,472	11,519
	81	1060	2121	3181	424I	5301	6362	7422	8,482	9,543	10,603	11,663
	8 ₂ 8 ₃	1073	2147	3220	4294	5367	6440	7514	8,587 8,692	9,660	10,734	11,807
	84	1087	2173	3259 3299	4346	5432 5498	6519	7605	8,797	9,778 9,896	10,305	12,000
	85	1113	2225	3338	4451	5563	6676	7780	8,901	10,014	11,127	12,235
	86	1126	2251	3377	4503	5629	6754	7880	0,006	10,132	11,257	12,383
	87	1130	2278	3417	4555	5694	6833	7972	0,111	10,249	11,388	12,527
	88	1152	2304	3456	4607	5760	6912	8063	9,215	10,367	11,519	
	80	1161	2330	3495	4660	5825	6990	8155	9,320	10,485	11,650	
	90	1178	2356	3534	4712	5891	7069	8247	9,425		11,780	
	91	1191	2382	3574	4765	5956	7147	8338	9,530	10,721	11,912	
	92	1204	2408	3613	4817	6021	7226	8430	9,634	10,839	12,043	
	93	1217	2435	3652	4870	6087	7304	8522	9,739	10,956	12,174	İ
	94	1231	2461	3692	4922	6152	7383	8613	9,844	11,074	12,305	
	95	1244	2487	3731	4974	6217	7461	8704	9,948	11,192	12,436	1
	96	1257	2513	3770	5027	6283	7540	8796	10,053	11,310	12,566	
	97	1270	2539	3809	5079	6349	7618	8888	10,158	11,428		1
	98	1283	2566	3849	5131	6414 6480	7697	8980	10,263	11,545		1
	99	1296	2592 2618	3888 3927	5183 5236	6545	7775 7854	9071	10,367	11,003		
	1-00	1309	2010	392/	3230	0343	1034	9103	10,4/2	11,701		1
et an and					<u></u>							

Revolutions per Minute

4,400	2,200	1,100	288	276	863	1,152	1,440	1,728	2,016	2,304	2,592	2,880	3,168	3,456	3,744	4,032	4,320	4,608	4,896	5,184	5,472	5,760	6,048	6,336	6,623
4,000	2,000	1,000	262	524	785	1,047	1,309	1,571	1,832	2,094	2,356	2,618	2,880	3,143	3,403	3,005	3,927	4,189	4,451	4,712	4,974	5,236	5,498	2,760	6,021
3,800	006,1	950	249	407	746	995	1,244	1,492	1,741	066'1	2,238	2,487	2,736	2,084	3,233	3,482	3,731	3,979	4,228	4,477	4,725	4,974	5,223	5,472	5,720
3,600	1,800	006	236	471	707	012	8/1/1	1,414	649,1	1,885	2,121	2,356	2,592	2,827	3,063	3,299	3,534	3,770	900,4	4,241	4,477	4,712	4,948	5,184	5,410
3,400	1,700	850	223	745	999	800	1,113	1,335	1,558	1,780	2,003	2,225	2,418	2,670	2,893	3,115	3,338	3,500	3,783	4,006	4,228	4,451	4,673	4,896	5,118
3,200	1,600	800	200	410	028	838	1,047	1,257	1,466	1,675	1,885	2,004	2,304	2,513	2,723	2,032	3,142	3,351	3,560	3,770	9,676	4,189	4,398	4,608	4.817
3,000	1,500	750	901	202	282	282	082	1.178	1,375	1.571	1,767	1,064	2.160	2,356	2,552	2,740	2,045	3,142	3,338	3,534	3,731	3,927	4,123	4,320	4.516
2,800	1,400	700	183	267	700	733	010	1.100	1.283	1.466	1.6.10	1.833	2.016	2,100	2,382	2,566	2,740	2,032	3.115	3,200	3,482	3,665	3.848	4.032	4.215
2,600	1,300	650	170	240	0 0	189	851	1.021	101.1	1,361	1.531	1.702	1.872	2,042	2,212	2,382	2.552	2,723	2.803	3.063	3,233	3,403	3,573	3,744	3.017
2,400	1,200	009	157		271	200	200	042	1.100	1.257	1.414	1.571	1228	1.885	2,042	2,100	2.356	2.513	2.670	2.827	2,085	3,142	3,200	3.456	3.613
		L	-		4 0	2 -	+ v	20	1	-00		, 5	: :	12	13	14	ĭ	10	17	- 82	101	20	21	22	22
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6,912 7,300 7,487	7,775	8,063	8,351	8.630	8,027	0.215	0.503	0,701	10,070	10,367	10,655	10,043							0 +11	nnc	2.600		1,300	650		10,380	10,551	10,721	10,891					
6,283 6,545 6,807	690,7	7,330	7,592	7,854	8,116	8,378	8.610	8,001	0,163	9,125	0,687	9,048	10,210	10,472	10,734	10,000			Danglutions non Minuto	um pd eno	2,400		1,200	900		9,552	9,739	0,590	10,053	10,210	10,367	10,524	10,681	10,839
5,969 6,218 6,466	6,715	6,963	7,213	7,461	7,710	7.050	8,207	8,456	8,705	8,054	0,202	9,451	002,6	8+6,6	10,197	10,416	10,695	10,013	Danoluti	Trevoine	CONTRACTOR OF THE PARTY OF THE	200			-	302	31 02	312 03	32 64	322 65	33 66	332 67	34 68	3+4 69
5,655 5,891 6,126	6,362	6,597	6,833	690,7	7,304	7,540	7,775	8,011	8,247	8,482	8,718	8,954	9,189	9,425	0,000	968,6	10,131	10,367	10,603	10,839					-	153	1.52	_	_	_	_	10# ID!		174
5,341 5,563 5,786	6,008	6,231	6,453	9,676	6,988	7,121	7,343	7,566	7,789	8,011	8,234	8,456	8,679	8,901	9,124	9,346	6,200	162.6	10,01	10,236	10.459	10,681	10,903											
5,027 5,236 5,445	5,655	2,864	6,074	6,283	6,493	6,702	6,612	7,121	7,330	7,540	7,749	7,959	8,168	8,378	8,587	8,797	900,6	9,215	9,425	9,634	9,844	10,053	10,263	10,472	10,031	10,501								
4,712 4,909 5,105	5,30 <u>I</u>	2,498	5,694	5,890	980,9	6,283	6,479	9/9/9	6,872	2,069	7,265	7,461	7,658	7,854	8,050	8,247	8,443	8,639	8,836	9,032	9,228	9,425	9,621	9.818	10,014	10,210	10,407	10,003	10,799	10,996				
4,398 4,581 4,764	4,948	5,131	5,314	5.498	5,681	5,864	8,0,9	6,231	6,414	6,297	6,78I	6,964	7,147	7,330	7,514	7,697	7,880	8,063	8,247	8,+30	8,013	8,797	8,980	9.163	9.340	9,529	9,712	060,6	10,079	10,263	0++,01	10,629	10,812	966,01
4,084 4,254 4,424	4,594	4,704	4,935	5,105	5,275	5,445	5,615	5,785	2,956	6,126	962'9.	99†,9	6,637	6,807	6,977	7,147	7,317	7,487	7,658	7,828	2,608	8,168	8,338	8.508	8,079	0,049	9,019	621,6	9,359	9,530	002,6	0,870	10,040	10,210
3,770 3,927 4,084	4,241	4,398	4,555	4,712	4,870	5,027	5,184	5,341	5,498	5,655	5,812	5,969	6,126	6,283	6,440	6,597	6,754	6,912	2,069	7,220	7,383	7,540	7,697	7,854	3,0II	0,100	0,525	20,402	8,039	8,797	8,954	9,111	9,268	9,425
25 26 26	27	28	50	30	31	32	33	34	35	36	37	38	39	9	41	42	43	++	45	40	47	48	49	20	51	22	55	54	22	20	57	258	20	9
12 12 12 13 13 13 13 13 13 13 13 13 13 13 13 13	54 I32	7 14	74 1.43	15	151 152				84 172	-	18 IS	-les e		-	mal with		200	22	222	23	232	24	242	HICAG	149	_		lear	272	28	282	29	203	30
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POWER REQUIRED BY MACHINE TOOLS

Engine	LATHES
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	Horse	-power
Swing	Average	Heavy
Inches	work	work
I 2	$\frac{1}{2}$	2
14	$\frac{3}{4}$ - I	2 - 3
16	I - 2	2 - 3
18	2 - 3	3 - 5
20-22	3	$7\frac{1}{2}$ -10
24-27	5	$7\frac{1}{2}$ -10
30	5 - 71	7 1 -10
32-36	$7\frac{1}{2}$ -10	10 -15
38–42	10 -15	15 -20
48-54	15 -20	20 -25
60-84	20 -25	25 -30

AXLE LATHES

	n.P.
Single	
Double	10-15 -20
Locomotive axle	25

WHEEL LATHES

Tailstock

		Motor
	H.P.	H.P.
48-in. car wheel	15-20	5
51-60 driv. wheel		5
79-84	25-30	5.
90	30-40	$5^{-7\frac{1}{2}}$
100	40-50	$5^{-7\frac{1}{2}}$
Quartering attach-		
ment	3- 5	

Cylinder Lathes

	H.P.
40 in	15
40-in. heavy	20
48 in	15

VERTICAL BORING MILLS

Size	Average H.P.	Heavy H.P.
36-42 in.	$5 \rightarrow 7\frac{1}{2}$	$7\frac{1}{2}$ -10
50 in.	$7\frac{1}{2}$	$7\frac{1}{2}$ -10
60-84 in.	$7\frac{1}{2}$ -10	10 -15
7-12 ft.	10 -15	30 -40
14-25 ft.	15 -25	30 -40

HORIZONTAL BORING, DRILLING AND MILLING MACHINE

Dia.	Horse-power
Spindle	for Single Spindle
$3\frac{1}{2}-4\frac{1}{2}$	$5 - 7^{\frac{1}{2}}$
$4\frac{1}{2} - 5\frac{1}{2}$	7 1 -10
$5\frac{1}{2} - 6\frac{1}{2}$	10 -15

For double spindle, use double the horse-power.

CYLINDER BORING MACH. Dia. Max. Boring Horse-power 4 20 7½ 6 30 10 8 40 15

MISCELLANEOUS

Loco. rod boring mach. $7\frac{1}{2}$ -10 Car-box boring mach. 6×12 and $5\frac{1}{2} \times 10''$ boxes $5 - 7\frac{1}{2}$ Car-wheel borer..... 10 -15

PLANERS

Size	Horse-power
24 x 24 in.	3-5
30 x 30 in.	$5-7\frac{1}{2}$
36 x 36 in.	10-15
48 x 48 in.	15-20
60 x 60 in.	20-25
72 x 72 in.	20-30
84 x 84 in.	30
100 x 100 in.	40

Note. — Normal length of bed in feet is about $\frac{1}{4}$ the width in inches.

FROG AND SWITCH PLANERS

36 x 12 in.	30
48 x 36 in.	30

PLATE PLANERS

Niles Nos. 2 and 3	10
Niles No. 5	15
Niles Nos. 6 and 7	20
Niles No. 8	20-25

ROTARY PLANERS	HORIZONTAL SLAB MILLERS
Dia. Cutter Horse-power	Width Between Horse-power
$24-30 \text{ in.}$ $5-7\frac{1}{2}$	Housings Average Heavy
36- 42 in. 10	$24-30$ in. $7\frac{1}{2}-10$ 10-15
48- 54 in. 15 60- 72 in. 20-25	36 in. 10 -15 20-25
84-100 in. 30-40	60-72 in. 25 75
64 100 m. 30 40	
Shapers	Cylindrical Grinders
H.P.	Dia. Wheel Average Heavy
12-16-in. stroke 2	
18-in. stroke 2 - 3	10 in. $5 7\frac{1}{2}$
20-24-in. stroke 3 - 5	14 in. 10 15 18 in. 10 15
30-in. stroke $5 - 7\frac{1}{2}$	18 in. 10 15
20-in. Traverse-head. $7\frac{1}{2}$ 24-in. Traverse-head. 10	77
24-iii. Traverse-nead 10	EMERY GRINDERS
Crank Slotters	No. Wheels Dia. H.P.
Stroke Horse-power	2 6 in. $\frac{1}{2}$ - 1
6- 8 3 - 5	2. 10 in. 2
10-12 5	2 12 in. 3
$13 14 5 - 7\frac{1}{2}$	$\frac{2}{18}$ in. $\frac{5}{7^{\frac{1}{2}}}$
$16-18$ $7\frac{1}{2}-10$	2 24-26 in. $7\frac{1}{2}$ -10
20-30 10 -15	
	Misc. Grinders
PLAIN MILLERS	Type Horse-power
Table Feed Cross Feed H.P.	Wet tool 2-3
34 IO $7\frac{1}{2}$	Flexible swing 3
42 12 10	Angle cock 3
50 12 15	Piston rod 3
Universal Millers	Twist drill
	0 1 1
Nos. $1-1\frac{1}{2}$ $1-2$ No. 2 $3-5$	Car wheel 30
No. 2	D ***
No. 4 $7\frac{1}{2}$ -10	Buffing Heads
No. 5 10 -15	No. Wheels Dia. H.P.
	$\frac{1}{4}$ 6 $\frac{1}{4}$ $\frac{1}{2}$
VERTICAL MILLERS	2 ' IO I -2
Height Under Work	2 14 3 -5
12-14 in. $5-7\frac{1}{2}$	
18 in. 10	Vertical Drills
20 in. 15	Size H.P.
24 in. 20	12-20 in
V C 3.5	24-28 in 2
VERTICAL SLAB MILLERS	30-32 in
24-in. width of work $7\frac{1}{2}$	36–40 in 5
32-36-in. width of work 10	50-60 in 5 -7½
42-in. width of work 15	Sensitive drills up to $\frac{1}{2}$ -in. $\frac{1}{4}$

RADIAL DRILLS	PIPE THREADING AND CUTTING
Horse-power	OFF MACHINES
Size Average Heavy	Size of Pipe Horse-power
3-ft. arm 1-2 3	$\frac{1}{4}$ 2 in. 2
4-ft. arm $2-3$ 5 - $7\frac{1}{2}$	$\frac{1}{2}$ - 3 in.
$5-6-7$ ft. arm $3-5$, $5-7\frac{1}{2}$	$1\frac{1}{4}$ 6 in. 3-5
8-9-10 ft. arm $5-7\frac{1}{2}$ $7\frac{1}{2}$ -10	2 - 8 in. $3-5$
MultiSpindle Drills	2 - 8 in. 3-5 3 -10 in. 5
	4 -12 in. 5
Size of Drills Up to H.P.	$8 - 18 \text{ in.}$ $7\frac{1}{2}$
$\frac{1}{3}\frac{1}{2} - \frac{1}{4}$ 6-10 spindle 3	24 in. 10
$ \begin{array}{c ccccccccccccccccccccccccccccccccccc$	Hammers
$\frac{1}{4}$ - $\frac{3}{4}$ 6-10 spindle 10	Size Horse-power
$\frac{3}{4}$ - 1 6-10 spindle 10-15	$15 - 75 \text{ lb.}$ $\frac{1}{2} - 5$
2 4 spindle $7^{\frac{1}{2}}$	100-200 lb. $5 - 7\frac{1}{2}$
GEAR CUTTERS	Drop hammers require approx-
Size Horse-power	imately i horse-power for every
36 x 9 in. 2 - 3	100-pound weight of hammer
48 x 10 in. 3 - 5	head.
30-60 x 12 in. $5 - 7^{\frac{1}{2}}$ 72 x 14 in. $7^{\frac{1}{2}}$ -10	100 lb. Bradley hammer 3
$72 \times 14 \text{ in.}$ $7\frac{1}{2}-10$	200 lb. Bradley hammer 5
64 x 20 in. 10-15	350 lb. Beaudry hammer 5
COLD SAWS	Bulldozers, Forming or
Dia. Saw Horse-power	Bending Machines
	Width Head Movement Horse-power
20 in. 3 26 in. 5	29 in. 14 in. 5
32 in. $7\frac{1}{2}$	34 in. 16 in. $7\frac{1}{2}$
36 in. 10-15	39 in. 16 in. 10
42 in. 20	34 in. 16 in. $7\frac{1}{2}$ 39 in. 16 in. 10 45 in. 18 in. 15
48 in. 25	63 in. 20 in. 20
Hacksaws ½	
BOLT CUTTERS	Bulldozers (ajax)
	No. 3
Single Horse-power	No. 4 $7\frac{1}{2}$
$I, I_{\frac{1}{4}}^{\frac{1}{4}}, I_{\frac{1}{2}}^{\frac{1}{2}} in. \dots I^{-2}$	No. 5 10 No. 6 15
$1\frac{3}{4}$, 2 in	
$2\frac{1}{2}$, $3\frac{1}{2}$ in	No. 7 30
4, 6 in $5-7\frac{1}{2}$ 1, $1\frac{1}{2}$ in. double $2-3$	No. 9 40
2, $2\frac{1}{2}$ in. double 3-5	No. 12 50
1, $1\frac{1}{2}$, 2 in. triple $3-7\frac{1}{2}$	BOLT HEADERS (HOT)
Bolt Pointers	Size Horse-power
	$\frac{3}{4}$ - $1\frac{1}{2}$ in. $5 - 7\frac{1}{2}$ 1 1 2 2 in. 10 - 15
$1\frac{1}{2}, 2\frac{1}{2}$ in	
NUT TAPPERS	Upsetting Machines
1, 2 in. 4-spindle 3	2 in. $7\frac{1}{2}$ -10
2 in. 6-spindle 3-5	3 in. 10 -15
2 in. 10-spindle 5	5 in. 15 -20
2 in. nut facer 3	6 in. 20 -30

TT - N - M) C		
Hot Nut Machi	PLATE SHEARS Metal Cut per Min. Stroke H.P.				
1	e-power	1	-		
$\frac{1}{2}$ - $\frac{5}{8}$ in. 5		3 X 24 11	0.	, , ,	n. Io
	12-10	1 x 24 ir		· ·	in. 15
$1\frac{1}{2}$ -2 in. 10	-15	2 X 14 ir			
Hyd. Wheel Pr	Tee	I X 42 II			n. 20
		1 1 X 42 ir			
	rse-power	1 1 x 54 ir			in. 75
100 tons	$\begin{array}{c} 5\\ 7\frac{1}{2} \end{array}$	1½ X 72 ir		04.	n. 10
200–300 tons		1 1 X 100 ir	1. 10-12	$7\frac{1}{2}$ i	in. 75
400 tons	10	т	EVER S	TEADE	
600 tons	15	Metal (e-power
BENDING AND STRAIG	GHTENING	IXI	in.		5
Rolls		$1\frac{1}{2} \times 1\frac{1}{2}$	in.		$7\frac{1}{2}$
Width Thickness	H.P.	2 X 2	in.		IO
4-6 in. $\frac{5}{16}$ - $\frac{3}{8}$	5	6 x 1	in.		10
6 in. $\frac{7}{16} - \frac{3}{4}$ 8 in. $\frac{7}{8}$	5-15	$2\frac{1}{2} \times 2\frac{1}{2}$	in.		10
8 in 7/8	25	1 x 7	in.		15
10 in. 1\frac{1}{8}-1\frac{1}{2}	35-50	$2\frac{3}{4} \times 2\frac{3}{4}$	in.		15
24 in. I	50	1½ x 8			20
		$3\frac{1}{2} \times 3\frac{1}{2}$			20
Flue Machine		4½ roui	nd		30
No. of Flue		Motors	USUALI	v E	ADT OVER
Flue Rattler Capacity	power		RANES A		MPLOYED
Flue Rattler 250–300 Flue Cutter	-		KANES A Ioist	IND III	01515
riue Cutter	2-3				
Flue Welder		Capacity		eed	H.P.
Flue Welder	2-3		Sp ft. pe	er min.	H.P.
		Capacity	Sp ft. pe	er min. 25	15
NOTCHING PRESS (SHE Dia. Punch Thickness F	2-3	Capacity tons 5	Sp ft. pe	er min. 25 50	15 25
Notching Press (she Dia. Punch Thickness F	2-3 EET-IRON)	Capacity tons	Sp ft. pe	er min. 25 50 30	15 25 25
NOTCHING PRESS (SHE Dia. Punch Thickness F $\frac{3}{8}$ in. $\frac{1}{4}$ $\frac{1}{2}$ $-\frac{5}{8}$ in. $\frac{1}{2}$ $-\frac{5}{8}$ in.	2-3 EET-IRON) Horse-power	Capacity tons 5	Sp ft. pe	er min. 25 50 30 40	15 25 25 40
Notching Press (she dispersion of the property of the press of the pressure o	2-3 EET-IRON) Horse-power	Capacity tons 5 10	Sp ft. pe	er min. 25 50 30 40	15 25 25 40 25
NOTCHING PRESS (SHE Dia. Punch Thickness F $\frac{3}{8}$ in. $\frac{1}{2}$ - $\frac{5}{8}$ in. $\frac{1}{2}$ - $\frac{5}{8}$ in.	2-3 EET-IRON) Horse-power I 2-3	Capacity tons 5 10 15	Sp ft. pe	er min. 25 50 30 40 20	15 25 25 40 25 25
Notching Press (she dispersion of the property of the press of the pressure o	2-3 EET-IRON) Horse-power I 2-3 3-5	Capacity tons 5 10	Sp ft. pe	er min. 25 50 30 40 20	15 25 25 40 25 25 25
Notching Press (she dispersion of the property of the propert	2-3 EET-IRON) Horse-power 1 2-3 3-5 5,	Capacity tons 5 10 15 20 25	Sp ft. pe	er min. 25 50 30 40 20 15	15 25 25 40 25 25 25 33
NOTCHING PRESS (SHE Dia. Punch Thickness F $\frac{3}{8}$ in. $\frac{1}{2} - \frac{5}{8}$ in. $\frac{1}{2} - \frac{5}{8}$ in. $\frac{3}{4}$ in. $\frac{3}{8} - 1$ in. $\frac{3}{4}$ in. $\frac{1}{2} - \frac{3}{4}$ I in. I	2-3 EET-IRON) Horse-power 1 2-3 3-5 5 7 1 7 2	Capacity tons 5 10 15 20 25	Sp ft. pe	er min. 25 30 30 40 20 15 10	15 25 25 40 25 25 25 25 33 33
NOTCHING PRESS (SHE Dia. Punch Thickness F $\frac{3}{8}$ in. $\frac{1}{4}$ $\frac{1}{2} - \frac{5}{8}$ in. $\frac{3}{4}$ in. $\frac{3}{4}$ in. $\frac{3}{8}$ in. $\frac{3}{4}$ in. $\frac{1}{2} - \frac{3}{4}$ I in. I $\frac{1}{4}$ in. I	2-3 EET-IRON) Horse-power 1 2-3 3-5 5 7 7 7 7 2-10	Capacity tons 5 10 15 20 25 30 5 au	Sp ft. pe	er min. 25 30 30 40 20 15 10	15 25 25 40 25 25 25 25 33 33 25
NOTCHING PRESS (SHE Dia. Punch Thickness F $\frac{3}{5}$ in. $\frac{1}{4}$ in. $\frac{1}{2}$ - $\frac{5}{8}$ in. $\frac{3}{4}$ in. $\frac{3}{4}$ in. $\frac{3}{4}$ in. $\frac{1}{4}$ in. I $\frac{1}{4}$ in. I $\frac{3}{4}$ in. I	2-3 EET-IRON) Horse-power 1 2-3 3-5 5 7 7 7 7 1 10 -15	Capacity tons 5 10 15 20 25 30 5 au 10 au	Sp ft. pe	er min. 25 50 40 20 15 10 14 50 22 25	15 25 25 40 25 25 25 25 33 33 25
NOTCHING PRESS (SHE Dia. Punch Thickness F $\frac{3}{8}$ in. $\frac{1}{2} - \frac{5}{8}$ in. $\frac{1}{2} - \frac{5}{8}$ in. $\frac{3}{4}$ in. $\frac{3}{4}$ in. $\frac{1}{2} - \frac{3}{4}$ in. $\frac{1}{1}$ in. I $\frac{1}{4}$ in. I $\frac{1}{4}$ in. I $\frac{3}{4}$ in. I	2-3 EET-IRON) Horse-power I 2-3 3-5 5 7 7 7 7 10-15 10-15 15-25	Capacity tons 5 10 15 20 25 30 5 au 10 au 50	Sp ft. pe	er min. 25 50 40 20 15 10 14 50 20 20 20 20 20 20 20 20 20	15 25 25 40 25 25 25 33 33 25 25
NOTCHING PRESS (SHE Dia. Punch Thickness F $\frac{3}{8}$ in. $\frac{1}{4}$ $\frac{1}{2} - \frac{5}{8}$ in. $\frac{1}{2} - \frac{5}{8}$ in. $\frac{1}{2} - \frac{5}{8}$ in. $\frac{1}{2} - \frac{5}{8}$ in. $\frac{1}{2} - \frac{1}{4}$ in. $\frac{1}{2} - \frac{3}{4}$ I in. I $\frac{1}{4}$ in. I $\frac{1}{4}$ in. I $\frac{1}{2}$ in. I	2-3 EET-IRON) Horse-power I 2-3 3-5 5 7 7 7 7 10-15 10-15 15-25	Capacity tons 5 10 15 20 25 30 5 au 10 au 50 5 au	Sp ft. pc ft. pc ft. xx. xx. xx. xx. xx. xx. xx. xx. xx. x	er min. 25 50 40 20 15 10 25 10 20 15 10 20 20 15 10 20 20 20 20 20 20 20 20 20	15 25 25 40 25 25 25 33 33 25 25 40 25
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NOTCHING PRESS (SHE Dia. Punch Thickness Fig. 1. Thickness Fig. 1	$2-3$ EET-IRON) Horse-power 1 $2-3$ $3-5$ 5 $7\frac{1}{2}$ $7\frac{1}{2}$ -10 $10-15$ $10-15$ $15-25$ CH e $7\frac{1}{2}$ -10	Capacity tons 5 10 15 20 25 30 5 au 10 au 50 5 au 10 au Capacity tons	x. x. x. x. x. Brid Span ft. 60 80	rr min. 25 50 30 40 20 15 10 10 25 10 25 25 25 26 20 25	15 25 25 40 25 25 25 33 33 25 25 40 25 25 Trolley H.P.
Notching Press (She Dia. Punch Thickness F \$\frac{3}{8}\$ in. \$\frac{1}{4}\$ \\ \frac{1}{2} - \frac{5}{8}\$ in. \$\frac{1}{2} - \frac{5}{8}\$ in. \$\frac{3}{4}\$ in. \$\frac{3}{4}\$ in. \$\frac{3}{4}\$ in. \$\frac{1}{2} - \frac{3}{4}\$ in. \$\frac{1}{2} - \frac{3}{4}\$ in. \$\frac{1}{2} - \frac{3}{4}\$ in. \$\frac{1}{2} - \frac{3}{4}\$ in. \$\frac{1}{2}\$ SHEARS Shears Gap Width Cut \$\frac{1}{8}\$ from: Gap Width Cut \$\frac{1}{8}\$ from:	2-3 EET-IRON) Horse-power I 2-3 3-5 5 7½ 7½ 10-15 10-15 15-25 CH e 7½-10	Capacity tons 5 10 15 20 25 30 5 au 10 au Capacity tons 5	x. x. x. x. x. x. x. x. x. x. 4. x.	er min. 25 50 40 20 15 10 15 10 25 10 25 10 25 10 20 21 20 20 20 20 20 20 20	15 25 25 40 25 25 25 33 33 25 25 40 25 25 Trolley H.P.
Notching Press (She Dia. Punch Thickness For Sin. 1/2 - 5/8 in. 1/2 - 5/8 in. 1/2 - 5/8 in. 1/2 - 5/8 in. 1/2 - 1/2 in. 3/8 plat Shears Gap Width Shears Cut 1/2 in. 3/56-60 in. 4/72-96 in. 5	2-3 EET-IRON) Horse-power 1 2-3 3-5 5 7 7 7 2-10 10-15 10-15 15-25 CH e 7 2-10 power Cut 1/2 Iron 5	Capacity tons 5 10 15 20 25 30 5 au 10 au 50 5 au 10 au Capacity tons 5 10	x. x. x. x. x. x. Sprid Span ft. 60 80 80 80 80	rr min. 25 50 30 40 20 15 10 10 25 10 25 25 25 26 20 25	15 25 25 40 25 25 25 33 33 25 25 40 25 25 Trolley H.P.
Notching Press (She Dia, Punch Thickness For D	2-3 EET-IRON) Horse-power 1 2-3 3-5 5 7 7 2-10 10-15 10-15 15-25 CH e 7 2-10 power Cut ¼ Iron 5 7 2 10 7 2 10 7 2 10	Capacity tons 5 10 15 20 25 30 5 au 10 au 50 5 au 10 au Capacity tons 5 10 15	x. x. x. x. x. Brid Span ft. 60 80 80 80 80 80	er min. 25 50 30 40 20 15 10 15 14 50 25 10 25 25 25 20 25 25	15 25 25 40 25 25 25 33 33 25 25 40 25 25 Trolley H.P.
Notching Press (She Dia, Punch Thickness For P	2-3 EET-IRON) Horse-power 1 2-3 3-5 5 7 7 7 2-10 10-15 10-15 15-25 CH e 7 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	Capacity tons 5 10 15 20 25 30 5 au 10 au 50 5 au 10 au Capacity tons 5 10 15 20	x. x. x. x. x. Span ft. 60 80 80 80 80 80 80 80 80	rr min. 25	15 25 25 40 25 25 25 33 33 25 25 25 25 40 25 25 25 33 33 33 5 5 5 5 5 5 5 5 7 7
Notching Press (She Dia, Punch Thickness For D	2-3 EET-IRON) Horse-power 1 2-3 3-5 5 7 7 7 2-10 10-15 10-15 15-25 CH e 7 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	Capacity tons 5 10 15 20 25 30 5 au 10 au 50 Capacity tons 5 10 15 20 25	x. x. x. x. x. Brid Span ft. 60 80 80 80 80 80	rr min. 225 500 330 400 200 155 144 500 225 225 225 225 225	15 25 25 40 25 25 25 33 33 25 25 40 25 25 Trolley H.P.

POWER REQUIRED FOR PLANING-MILL EQUIPMENT

В	AND SAW	S	Size	Heads	Horse-power
	Max. wio	lth	30 x 8 in.	4-5	40
Dia. Wheel	of Saw		24 X 12 in.	4-5	40
30 in.	$\frac{1}{2}$	2	30 x 12 in.	4-5	40
34 in.	$\frac{\overline{1}}{2}$	3	0	·) (
36-38 in.	$\frac{1}{2} - 1\frac{\bar{1}}{2}$	3-5	1	SIDE MOU	
40-42 in.	$\frac{1}{2} - 1\frac{1}{2}$	$3^{-7\frac{1}{2}}$	Capacity		Horse-power
40-42 in.	$2\frac{1}{2}$	10	4 x 4 in.	1-2	5_
40-42 in.	$3\frac{1}{2}$	15	4 x 4 in.		$7\frac{1}{2}$
• •	-	·	6 x 4 in.	1-2	5_
Cu	T OFF SA	WS	6 x 4 in.	3-4	$7\frac{1}{2}$
Dia. Saw	No. of Sav	vs H.P.	8 x 4 in.	4	10
12-14	1	3	10 x 4 in.	4	15
16	1	5	12 x 5 in.	4	20
16	2	$7\frac{1}{2} - 10$	14 x 6 in.	4	20
30	1	$7\frac{1}{2}$	TNOT	DE MOUL	DEDC
•					
Circu	LAR RIP	Saws	8 x 4 in.	4,	15
14	1	5	10 x 4 in.	4	15
16	1	$\frac{5}{7^{\frac{1}{2}}}$	10 x 6 in	4-5	20-30
24	1	10	15 x 4 in.	4-5	20-30
36	ī	15		JOINTERS	3
7	_	•	8-12	in.	2
Tin	iber Size	RS	16-24	in.	3
Capacity	No. Head	s H.P.	30-36		5
30 x 20 in.	4	50	"		·
20 x 20 in.	4	50	TENO	NING MA	CHINES
30 x 10 in.	4	40		No. Heads	
20 x 16 in.	4	40	$5\frac{1}{2} \times 14 \text{ in.}$	I	3 - 5
	•	, -	$5\frac{1}{2} \times 15$ in.	. 2	5
S	URFACERS	3	23 x 9 in.		$7\frac{1}{2}$
Size No	o. Heads	H.P.	$54 \times 4\frac{1}{2}$ in	ı. 4–8	10 -15
30 x 6 in.	1-2	15-20 J	$78 \times 4\frac{1}{2}$ in	1. 4-8	10 -15
24 x 6 in.	1-2	15-20 ≱	Gainers		$7\frac{1}{2}$ -10 -15
30 x 8 in.	2	30 30 Heavy	R	ELT SANDI	PDC
26 x 8 in.	2	30 H	Width of Belt		
24 x 6 in.	_	1) _	6-14 in.		Horse-power
16 x 6 in.	I	$ \begin{array}{c} \begin{array}{c} $	18 in.		2-3
24 x 8 in.	I	5-71	10 111.		5
30 x 8 in.	I	IO H	C	OLUMN AI	RM
30 x 8 III.	I		Length of	Dia. of	
		Horse-power	Arm	Disks	Horse-power
Shapers, 1-2			4-10 ft.	8 in.	3
Borers		$5^{-7\frac{1}{2}}$	D-	RUM SAND	- nna
DIAMERO	Матсн	ODG AND			
	ING MAC		Length of Dru		Horse-power
			30 ii	n.	$7^{\frac{1}{2}}$
Size	Heads	•	36 ii		10
9 x 8 in.	4-5	30	42-48 ir	1.	15

30

40

4-5

19 x 8 in.

24 x 8 in.

54-60 in.

72-84 in.

20

30

GROUP DRIVING OF MACHINES

There are many shops where group driving will be found more desirable than the use of individual motors, both as to first cost and maintenance. This is particularly true where the machines are comparatively small and run intermittently, as the cost of motors will be much less.

Friction load of $2\frac{1}{2}$ to 3-inch shafting, with bearings 8 to 10 feet and running at 150 to 200 revolutions per minute, is about 1 horsepower for every 30 feet of shafting. This includes the friction of

countershafts of the machines driven by it.

In group driving it is usually perfectly safe to select a motor having a rated capacity of from 25 to 30 per cent. of the total power required for the machines in the group.

POWER REQUIRED FOR PUNCHING AND SHEARING

Experiments tend to show that with steel plates of 60,000 pounds tensile strength, the metal is all sheared when the punch has passed $\frac{1}{3}$ through the plate. The following formula by L. R. Pomeroy takes this into account and also allows the motor and efficiency of 80 per cent. and the punching machine 75 per cent.

When

T = Full thickness of plate.

D = Diameter of hole punched.

N = Number of holes punched per minute.P = Horse-power required to drive machine.

$$P = \frac{T^2 \times D \times N}{3.78}$$

Taking a $\frac{1}{2}$ -inch hole in a $\frac{1}{2}$ -inch plate, the power required to punch 30 per minute would be

$$\frac{\frac{1}{4} \times \frac{1}{2} \times 30}{3.78} = \frac{3.75}{3.78}$$
 or about 1 horse-power.

Pressure required for shearing = Length of cut \times thickness in inches \times shearing strength of material. Dies with "shear" reduce this $\frac{1}{3}$ to $\frac{1}{2}$.

Power Required to Remove Metal

The power required to remove metal depends on the amount of metal removed per minute and the nature of the cutting tool. With a cutting angle of 75 to 80 degrees, tests show that for mild steel of 40-point carbon one horse-power will remove 1.5 cubic inches of metal per minute.

For average conditions and with tools as ordinarily used, tests show that to remove one cubic inch of metal per min. requires the

amount of power shown in the table.

Brass and similar alloys		H.P.
Wrought iron	0.6	"
Mild steel (0.30% to 0.40% carbon)		
Hard steel (0.50% carbon)		
Very hard tire steel	1.50	

Two important factors enter into the problem of power for driving machines. These are the Time Factor and the Load Factor.

 $Time Factor = \frac{Actual Cutting Time}{Total Time to Complete Operation}$

 $Load Factor = \frac{Average Daily Load}{Full Load Rating of Motor}$

Many tests give the following load factors:

The average load factor for motors driving lathes is from 10 to 25 per cent. On some special machines, as driving wheel and car wheel lathes, the cuts are all heavy, which increases the average

load factor to from 30 to 40 per cent.

For extension boring mills, 5 horse-power motors are used to move the housings on from 10 feet to 16 feet mills, $7\frac{1}{2}$ horse-power for from 14 feet to 20 feet mills and 10 horse-power for from 16 feet to 24 feet mills. The load factor of the driving motor on boring mills averages from 10 to 25 per cent.

The load factor of motor-driven drills is about 40 per cent. when the larger drills applicable thereto are used. If the smaller drills

are used the load factor averages 25 per cent. and lower.

For the average milling operations the load factor averages from 10 to 25 per cent. On slab milling machines where large quantities of metal are removed it will average from 30 to 40 per cent.

The work on this class of machinery is usually light and much time is required in making adjustments. Hence the load factor is

rarely higher than 20 per cent.

On planers the load factor averages between 15 and 20 per cent. The motor must be large enough to reverse the table quickly, yet this peak load occurs for such short intervals that it does not increase the average load per cycle very much.

The work done on shapers is of a varying character. With light work the load factor will not exceed from 15 to 20 per cent.; with

heavy work, the load factor will be as high as 40 per cent.

The conditions with slotters are similar to those on shapers.

Horse-Power to Drive Machines

Extensive experiments by L. R. Pomeroy show that the horsepower required equals the Feed per rev. or stroke X Depth of cut in inches X Cutting speed in feet per minute X 12 X Number of tools cutting X a Constant which depends on the material. This checks up fairly well with actual motor tests. The constants given are:

Cast iron	0.35 to 0.5
Wrought iron or soft steel	0.45 to 0.7
Locomotive driving wheel tires	0.70 to 1.00
Very hard steel	1.00 to 1.10

Handling this in another way, Charles Robbins of the Westinghouse Electric & Manufacturing Co. gives: The horse-power = Cubic inches removed per minute X a Constant. These constants are:

Brass and similar alloys	o.2 to o.3
Cast iron	
Wrought iron	0.6
Mild steel (0.30 to 0.40 carbon)	0.6
Hard steel (0.50 carbon)	1.00 to 1.25
Very hard tire steel	1.50

These represent average conditions with the cutting tools ordinarily used.

A brief summary of the studies by Mr. Robbins gives interesting data on various machines. These give factors as follows:

	Time Factor	Load Factor
Vertical boring machine Radial drilling machine Portable milling machine Portable slotting machine Planers	41 54	27% 10 55 12

L. R. Pomeroy also gives a method of determining the horse-power required by the belt used to drive the machine. The formula is:

Hp. = Thickness of belt in inches × Width of belt in inches × Diameter of pulley in inches × Revolutions per minute × Constant for kind of belt.

These constants are:

Leather belto.oo62 t	0 0.0098
Cotton belt	0.0068
Rubber belto.oo50 t	0 0.0082

STEEL AND OTHER METALS

HEAT TREATMENT OF STEEL

THE theory of the heat treatment of steel rests upon the influence of the rate of cooling on certain molecular changes in structure occurring at different temperatures in the solid state. These changes are of two classes, critical and progressive; the former occur periodically between certain narrow temperature limits, while the latter proceed gradually with the rise in temperature, each change producing alterations in the physical characteristics. By controlling the rate of cooling, these changes can be given a permanent set, and the physical characteristics can thus be made different from those in the metal in its normal state.

The highest temperature that it is safe to submit a steel to for heat-treating is governed by the chemical composition of the steel. Pure carbon steel should be raised to about 1650 degrees Fahr., while some of the high-grade alloy steels may safely be raised to 1750 degrees Fahr., and the high-speed steels may be raised to just below the melting point, usually from 2000 to 2150 degrees Fahr. It is necessary to raise the metal to these points so that the active cooling process will have the desired effect of checking the crystallization of the structure.

Methods of Heating

Furnaces using solid fuel such as coal, coke, charcoal, etc., are the most numerous and have been used the longest. These furnaces consist of a grate to place the fuel on, an arch to reflect the heat and a plate to put the pieces on. The plate should be so arranged that the flames will not strike the pieces to be heated, and for that reason some use cast-iron or clay retorts which are open on the side toward the doors of the furnace.

Liquid fuel furnaces, which have open fires and which use liquid fuels, are not very numerous at present, but their use is increasing, owing to the ease with which the fire is handled and the cleanliness

as compared with a coal, coke or charcoal fire.

Crude oil and kerosene are the fuels generally used in these furnaces, owing to their cheapness and the fact that they can be easily obtained. These fuels are usually stored in a tank near the furnaces and are pumped to them or flow by force of gravity.

Heating in Liquids

Furnaces using liquid for heating have a receptacle to hold the liquid, which is heated by coal, oil, gas or any other economical means; the liquid is kept at the highest temperature to which the piece should be heated. The piece should be heated slowly in an ordinary furnace to about 800 degrees, after which it should be

immersed in the liquid bath and kept there long enough to attain the temperature of the bath and then removed to be annealed or hardened.

The bath usually consists of lead, although antimony, cyanate of potassium, chloride of barium, a mixture of chloride of barium and chloride of potassium in the proportion of 3 to 2, mercury, common

salt and metallic salts have been successfully used.

This method gives good results, as no portion of the piece to be treated can reach a temperature above that of the liquid bath; a pyrometer attachment will indicate exactly when the piece has arrived at that temperature, and its surface cannot be acted upon chemically. The bath can be maintained easily at the proper tem-

perature and the entire process is under perfect control.

When lead is used it is liable to stick to the steel unless it is pure and retard the cooling of the spots where it adheres. Impurities, such as sulphur, are liable to be absorbed by the steel and thus affect its chemical composition. With high temperatures lead and cyanate of potassium throw off poisonous vapors which make them prohibitive, and even at comparatively low temperatures these vapors are detrimental to the health of the workmen in the hardening room. The metallic salts, however, do not give off these poisonous vapors, and are much better to use for this purpose, but many times the fumes are unbearable.

Gas as Fuel

Furnaces using gaseous fuel are very numerous and are so constructed that they can use either natural gas, artificial gas, or producer gas. They are very easy to regulate and if well built are capable of maintaining a constant temperature within a wide range.

In first cost this style of furnace is greater than that of the solid fuel furnaces, but where natural or producer gas is used the cost of operating is so much less that the saving soon pays for the cost of installation. Illuminating gas, however, is more expensive than the solid fuels and is only used where high-grade work demands the best results from heat treatment.

COOLING THE STEEL

Cooling apparatus is divided into two classes — baths for hard-

ening and the different appliances for annealing.

The baths for quenching are composed of a large variety of materials. Some of the more commonly used are as follows, being arranged according to their intensity on 0.85 per cent. carbon steel: Mercury; water with sulphuric acid added; nitrate of potassium; sal ammoniac; common salt; carbonate of lime; carbonate of magnesia; pure water; water containing soap, sugar, dextrine or alcohol; sweet milk; various oils; beef suet; tallow; wax. These baths, however, do not act under all conditions with the same relative intensity, as their conductivity and viscosity vary greatly with the temperature.

With the exception of the oils and some of the greases, the quenching effect increases as the temperature of the bath lowers. Sperm and linseed oils, however, at all temperatures between 32 and 250 degrees Fahr., act about the same as distilled water at 160 degrees.

The baths for hardening which give the best results are those in which some means are provided for keeping the liquid at an even temperature. Where but few pieces are to be quenched, or a considerable time elapses between the quenching of pieces, the bath will retain an atmospheric temperature from its own natural radiation. Where a bath is in continuous use, for quenching a large number of pieces throughout the day, some means must be provided to keep the temperature of the bath at a low even temperature. The hot pieces from the heating furnace will raise the temperature of the bath many degrees, and the last piece quenched will not be nearly as hard as the first

Annealing

The appliances for annealing are as numerous as the baths for quenching, and where a few years ago the ashes from the forge were all that were considered necessary for properly annealing a piece of steel, to-day many special preparations are being manufactured and

sold for this purpose.

The more common materials used for annealing are powdered charcoal, charred bone, charred leather, slacked lime, sawdust, sand, fire clay, magnesia or refractory earth. The piece to be annealed is usually packed in a cast-iron box, using some of these materials or combinations of them for the packing, the whole is then heared in a furnace to the proper temperature and set aside, with the cover left on, to cool gradually to the atmospheric temperature.

For certain grades of steel these materials give good results; but for all kinds of steels and for all grades of annealing the slow-cooling furnace no doubt gives the best satisfaction, as the temperature can be easily raised to the right point, kept there as long as necessary, and then regulated to cool down as slowly as is desired. The gas, oil or electric furnaces are the easiest to handle and regulate.

The Hardening Bath

In hardening steels the influence of the bath depends upon its temperature, its mass and its nature; or to express this in another way, upon its specific heat, its conductivity, its volatility and its viscosity. With other things equal, the lower the temperature of the bath, the quicker will the metal cool and the more pronounced will be the hardening effect. Thus water at 60 degrees will make steel harder than water at 150 degrees, and when the bath is in constant use the first piece quenched will be harder than the tenth or twentieth, owing to the rise in temperature of the bath. Therefore if uniform results are to be obtained in using a water bath, it must either be of a very large volume or kept cool by some mechanical means. In other words, the bath must be maintained at a constant temperature.

The mass of the bath can be made large so no great rise in temperature is made by the continuous cooling of pieces, or it can be made small and its rise in temperature used for hardening tools that are to remain fairly soft, as, if this temperature is properly regulated, the tool will not have to be re-heated and tempered later, and cracks and

fissures are not as liable to occur.

Another way of arriving at the same results would be to use the double bath for quenching, that is, to have one bath of some product similar to salt which fuses at 575 degrees Fahr. Quench the piece in that until it has reached its temperature, after which it can be quenched in a cold bath or cooled in the air.

BATH FOR DRAWING TEMPER

A VERY good table from which to make up baths for drawing the temper is as follows:

Composition of Bath Lead and Tin	Melting Point in degree F.	Color of Steel at Temperature Given
148	420	very faint yellow
158	430	.faint yellow
ı68	440	. light straw
178		
18.58	460	.full straw
20 8	470	.dark straw
248		
288	490	. brown
388	510	brown with purple spots
бо 8		
968	550	. deep purple
2008		
Boiling linseed oil.		
Melted lead	610	gray blue

These are used in a similar manner to the hardening baths, selecting the bath which gives the proper drawing temperature.

HIGH-SPEED STEELS

THESE steels are made by alloying tungsten and chromium or molybdenum and chromium with steel. These compositions completely revolutionize the points of transformation. Chromium, which has a tendency to raise the critical temperature, when added to a tungsten steel, in the proportions of 1 or 2 per cent., reduces the critical temperature to below that of the atmosphere. Tungsten and molybdenum prolong the critical range of temperatures of the steel on slow cooling so that it begins at about 1300 degrees Fahr. and spreads out all the way down to 600 degrees.

These steels are heated to 1850 degrees for the molybdenum and 2200 degrees for the tungsten, and cooled moderately fast, usually in and air blast, to give them the property known as "red-hardness." This treatment prevents the critical changes altogether and preserves the steel in what is known as the austenitic condition. The

austenitic condition is one of hardness and toughness.

One rule which has given good results in heat-treating these high-speed steels is to heat slowly to 1500 degrees Fahr., then heat fast to 2200 degrees; after which cool rapidly in an air blast to 1550 degrees; then cool either rapidly or slowly to the temperature of the air. Others advocate cooling in crude oil.

CASE-HARDENING

CASE-HARDENING, carbonizing, or, as it is called in Europe, "cementation," is largely used so that the outer shell can be made hard enough to resist wear and the core of the piece can be left soft enough to with-

stand the shock strains to which it is subjected.

Several methods different from the old established one of packing the metal in a box filled with some carbonizing material, and then subjecting it to heat, have been devised in the last few years. Among them might be mentioned the Harveyizing process which is especially applicable to armor plate. The Harveyizing process uses a bed of charcoal over the work, the plates being pressed up against it in a pit or furnace and gas turned on so that the steel will be heated through the charcoal, thus allowing the carbon to soak in from the top.

The result of the carbonizing operation is determined by five factors, which are as follows: First, the nature of the steel; second, the nature of the carbonizing material; third, the temperature of the carbonizing furnace; fourth, the time the piece is submitted to the carbonizing process; fifth, the heat treatment which follows carbonizing.

The nature of the steel has no influence on the speed of penetration of the carbon, but has an influence on the final result of the

operation.

If steel is used that has a carbon content up to 0.56 per cent., the rate of penetration in carbonizing is constant; but the higher the carbon content is, in the core, the more brittle it becomes by prolonged annealing after carbonizing. Therefore it is necessary that the carbon content should be low in the core, and for this reason a preference is given to steels containing from 0.12 to 0.15 per cent. of carbon for carbonizing or case-hardening purposes.

TABLE I. — PENETRATION OF CARBON PER HOUR WITH

Different Alloys	S			of Pene-
Component of Alloys	ī	t: Tr	ration	on per Inches
o.5 per cent. manganese				0.043
1.0 per cent. manganese				0.047
1.0 per cent. chromium				0.039
2.0 per cent. chromium				0.043
2.0 per cent. nickel				0.028
5.0 per cent. nickel				0.020
o.5 per cent. tungsten				0.035
1.0 per cent. tungsten				0.036
2.0 per cent. tungsten				0.047
o.5 per cent. silicon				0.024
1.0 per cent. silicon				0.020
2.0 per cent. silicon				0.016
5.0 per cent. silicon				0.000
i.o per cent. titanium				0.032
2.0 per cent. titanium				0.028
1.o per cent. molybdenum				0.036
2.0 per cent. molybdenum				0.043
1.0 per cent. aluminum				0.016
3.0 per cent. aluminum				0.008

The rate of penetration for ordinary carbonizing steel under the same conditions would have been 0.035 inch. Thus it will be seen that manganese, chromium, tungsten and molybdenum increase the rate of penetration. These seem to exist in the state of a double carbide and release a part of the cementite iron.

Nickel, silicon, titanium and aluminum retard the rate of penetration — 5 per cent. of silicon reducing it to zero — and these exist in

the state of solution in the iron.

The Carbonizing Materials

The nature of the carbonizing materials has an influence on the speed of penetration and it is very essential that the materials be of a known chemical composition as this is the only way to obtain like

results on the same steel at all times.

These materials or cements are manufactured in many special and patented preparations. The following materials are used and compounded in these preparations, but many of them give as good results when used alone as when compounded with others in varying percentages: Powdered bone; wood charcoal; charred sugar; charred leather; cyanide of potassium; ferro-cyanide of potassium; bichromate of potassium; animal black, acid cleaned. Prussiate of potash, anthracite, mixture of barium carbonate, graphite, petroleum gas, acetylene, horn, etc.

Wood charcoal is very largely used in carbonizing steels, but the value of this material varies with the wood used, the method employed in making the charcoal, and other factors. Used alone it gives the normal rate of penetration for the first hour, but after that the rate gradually decreases until at eight hours it gives the lowest rate of penetration of any of the carbonizing materials. The best

wood charcoal is that made from hickory.

Powdered charcoal and bone give good results as a carbonizing material and are successfully used in carbonizing nickel-chrome steel by packing in a cast-iron pot and keeping at a temperature of about 2000 degrees Fahr. for four hours, and then cooling slowly before taking out of the pot or uncovering.

TABLE 2

	MATERIALS	USED AND RATE	OF PENETRATIO	N IN INCHES
Temperature in Degrees Fahrenheit	Charcoal 60 per cent. + 40 per cent. of Carbon- ate of Borium	Ferro-cyanide 66 per cent. + 34 per cent. of Bichromate	Ferro-cyanide Alone	Powdered Wood Charcoal Alone
1300				
1475	0.020	0.033	0.020	0.020
1650	0.088	0.069	0.079	0.048
1825	0.128	0.128	0.128	0.098
2000	0.177	0.177	0.198	0.138

The speed of penetration caused by the action of different cements at different temperatures for the same time, *i.e.*, eight hours, is best shown by Table 2.

The nature of the carbonizing material has a very pronounced effect on the rate of carbonization, or the percentage of the carbon content in the surface layer of the piece, or both.

Another Test of Penetration

At the same temperature, i.e., 1825 degrees Fahr., for different lengths of time and with different cements, the rate of penetration

obtained was according to Table 3.

Eighty per cent. charcoal + 20 per cent. carbonate of barium, 40 per cent. charcoal + 60 per cent. carbonate of barium, ferrocyanide alone and 66 per cent. ferro-cyanide + 34 per cent. bichromate were used with practically the same results for eight hours' time.

TABLE 3

	Materials Used and Rate of Penetration in Inches				
Length of Time in Hours	Carbon 60 per cent. + 40 per cent. of Carbonate	Ferro-cyan- ide 66 per cent. + 34 per cent. of Bichromate	Powdered Wood Charcoal Alone	Charcoal and Carbonate of Potassium	Unwashed Animal Black
1	0.031	0.033	0.028	0.059	0.035
2	0.039	0.037	0.053	0.078	0.059
4	0.047	0.049	0.063	0.094	0.088
6	0.078	0.074	0.072	0.011	0.106
8	0.118	0.128	0.098	0.138	0.128

Another set of tests was carried out for a longer period of time, with other materials and at a uniform temperature of 1650 degrees Fahr., with the results given in Table 4.

TABLE 4

Toward of Winner	ETRATION IN INCHES		
Length of Time in Hours	Charred Leather	Ground Wood Charcoal	Barium Carbonate and Wood Charcoal
2	0.045	0.028	0.055
4	0.062	0.042	0.087
8	0.080	0.062	0.111
I 2	0.110	0.070	0.125
	1		1

In the use of hydrocarbons, or gases, a fresh supply can be kept flowing into the carbonizing receptacle and the time greatly reduced for deep penetration with an appreciable reduction of time for the

shallow penetrations.

The constitution of a given steel is not the same in the hardened as in the normal state, owing to the carbon not being in the same state. In the annealed or normal steel it is in a free state, while in a hardened steel it is in a state of solution which we may call martensite; and this contains more or less carbon according to the original carbon content of the steel. The composition, and therefore the mechanical properties, depend principally upon the carbon content, the mechanical properties being changed slowly and gradually by an increase in carbon.

This is best shown by Table 5 in which it will be seen that the tensile strength and elastic limit gradually increased with the increase in the percentage of carbon, both in the annealed and hardened state

Table 5. — Effect of Composition and Hardening on the Strength

	Case Harden- ing Steel		Low Carbon	Mediu Carbo		Very Carbon	
				_			
Carbon	0.10	0.14	0.23	0.52	0.60	0.72	
Silicon	0.00	0.05	0.15	0.18	0.10	0.17	
Manganese	0.10	0.33	0.45	0.35	0.40	0.38	
Phosphorus	0.016	0.023		0.02		0.03	
Sulphur	0.025	0.052	0.062	0.043)	0.06	
				"			
MECHAN	ICAL PRO	PERTIE	S WHEN	ANNEA	LED		
Tensile Strength (in pounds per square inch) Elastic Limit (in		61,500	66,500	97,800	116,400	130,700	
pounds per square inch)	36,300	35,200	41,200	52,600	66,500	75,800	
age in 4 inches)		27	26	20	14	9	
MECHANICAL PROPERTIES WHEN HARDENED							
Tensile Strength (in							

Tensile Strength (in pounds per square						
pounds per square inch)	66,400	73,100	99,400	132,100	153,400	180,100
Elastic Limit (in						
pounds per square						
inch)		39,600	54,000	81,400	102,100	105,500
Elongation (percent-						_
age in 4 inches)	24	22	14	9	4	0

while the elongation gradually decreased. These tests were made with bar ½ inch in diameter and 4 inches in length. It will also be seen that there was considerable change in the steels which were too low in carbon to be made so hard that they could not be filed. The reduction in elongation when the test bars were heated and quenched show that the metal was harder than when in the annealed state.

Selecting the Proper Temperature for Quenching

A hardening process that will produce a steel that is as homogeneous as possible is always sought for in practice. This is easily obtained in a high-carbon steel and especially if it contains 0.85 per cent. carbon, by passing the recalescent point before quenching. The desired homogeneity is not so easily obtained, however, in the low-carbon steels as they have several points of transformation. If these are quenched at a point a little above the lowest point of transformation the carbon will pass into solution, but the solution is not homogeneous. To obtain this result it is necessary that the quenching be done from a little above the highest point of transformation. This is higher in the low- than in the high-carbon steels. In practice this calls for a quenching of the low-carbon steels as about 1650 degrees Fahr., while a high-carbon steel should be quenched at about 1450 degrees.

Testing Pyrometers

Pyrometers can be tested by placing some common salt in an iron box and heating until it melts. Put the pyrometer in the molten salt and, if correct, it will register 1441 degrees Fahr.

A Table of Fahrenheit and Centigrade thermometer scales is given

on page 455.

Test of Hardness

The hardness of metals, particularly of steels which are heat treated, is now tested with either the Shore Scleroscope or the Brinnell Ball method.

THE BRINNELL TEST

The Brinnell method of testing consists of forcing a hardened steel ball of given dimension into the metal to be tested under a given pressure. The diameter of the impression made is read with a graduated microscope and the hardness found by consulting the table below. In this the ball is 10 millimeters in diameter. If, with a pressure of 3000 kilograms as indicated by the testing machine, the diameter of the depression is 3 millimeters, the hardness number is 418. Dividing this by 6 gives practically 70, as shown under the second column. According to this table, a pressure of only 500 kilograms will give a direct reading which is about the same as that of the Scleroscope. The standard pressure however is 3000 kilograms.

Table of Brinnell Hardness Numerals—steel ball of 10 millimeters diameter

Hardness Numeral Pressure. Kilograms	280	6.5.5.4.4.4.4.6.6.6.6.6.6.6.6.6.6.6.6.6.	12.2 11.9 11.7 11.5 11.3
Hard Num Press Kilog	3000	95.0 97.0 97.0 97.0 98.0 98.0 98.0 98.0 98.0 98.0 98.0 98	73.0 71.5 70.0 69.0 68.0
Diameter of Impres- sion, Milli-	meters	6.000000000000000000000000000000000000	6.75 6.85 6.95 6.95
Hardness Numeral Pressure, Kilograms	200	23.8 23.8 22.3.8 22.2.8 22.1.6 22.1.6 20.0.0 10.0.0 10.0.0 10.0.0 10.0.0 10.0.0	17.5 17.2 16.9 16.6
Harc Nun Pres Kilog	3000	143 140 137 137 128 126 126 118 118 116 1112	105 103 101 99
Diameter of Impres- sion, Milli-		\$	5.8.8 5.8.8 5.95 5.95 8.95
Hardness Numeral Pressure, Kilograms	200	3 3 3 3 5 0 0 0 3 3 5 5 0 0 0 0 0 0 0 0	25.9 25.9 24.9 24.4
Harc Nun Pres Kilog	3000	222 223 2217 2017 2007 1006 1008 1709 1709 1709 1709 1709 1709	156 156 153 149 146
Diameter of Impres- sion, Milli-		4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4 4	4.4.4.4.85 0.8.4.4.9.0.0.0.0.0.0.0.0.0.0.0.0.0.0.0.0.0
Hardness Numeral Pressure, Kilograms	200	070 070 073 073 074 075 074 074 074 074 074 074 074 074 074 074	44 44 40 30 39
Hard Nun Press Kilog	3000	418 402 387 364 364 351 332 332 332 332 332 263 269 269	252 253 248 241 235
Diameter of Impres- sion, Milli-		6. 6. 6. 6. 6. 6. 6. 6. 6. 6. 6. 6. 6. 6	3.75 3.80 3.85 3.90
Hardness Numeral Pressure, Kilograms	200	158 143 136 136 136 119 119 110 100 100 100 100 100 100 100	83 80 77 74
Harc Nun Pres Kilog	3000	946 898 8857 782 744 713 663 663 663 673 673 878 878 878 878 878	495 477 460 444 430
Diameter of Impres- sion, Milli- meters		44444444444444444444444444444444444444	2.75 2.80 2.90 2.95

SCLEROSCOPE READING

In the Shore Scleroscope, a miniature drop hammer tup falls from a fixed height to the surface of the metal being tested. The height of the rebound indicates the hardness on an arbitrary scale which has 115 divisions, these meeting all usual requirements. This method can be applied to any material which will take a permanent set under impact. For, no matter how hard the material, the falling weight, weighing about 40 grains, makes a dent which can be seen with a glass.

The following table shows the readings which will be obtained on the Scleroscope for the materials indicated, this giving the comparative hardness of the materials.

SCLEROSCOPE HARDNESS SCALE

Metal	Annealed	Hammered
Lead — cast	2- 5 4- 9 5 6 ¹ / ₂	3- 7 8½
Silver Brass — cast Pure Tin — cast Brass — drawn	7-35 8	20- 30 12 20- 45
Bismuth — cast	10	17 14- 20 20
Iron — pure Mild steel, o.15 carbon Nickel Anode — cast	18 22 31	25- 30 30- 45 55
Iron, gray — cast. Iron, gray — chilled Steel, tool, 1% carbon. Steel, tool, 1.65% carbon	30-35	50- 90 40- 50
Vanadium steel	35 ⁻ 45 47	· 60- 95 70-105 90-110

NOTE. — These figures vary with the composition and density of the metals. They are about $\frac{1}{6}$ those of the Brinnell test for equal hardness, varying somewhat with the kind of metal.

FAHRENHEIT AND CENTIGRADE THERMOMETER SCALES

F	С	F	С	F	С	F	С	F	С
- 40	- 40.	70	21.1	185	85.	950	510.	2100	1149.
- 35	- 37.2	75	23.9	190	87.8	1000	537.8	2150	1176.5
- 30	- 34.4	80	26.7	195	90.6	1050	565.5	2200	1204.
- 25	- 31.7	85	29.4	200	93.3	1100	593.	2250	1232.
- 20	- 28.9	90	32.2	205	96.1	1150	621.	2300	1260.
- 15	- 26.1	95	35.	210	98.9	1200	648.5	2350	1287.5
- 10	- 23.3	100	37.8	212	100.	1250	676.5	2400	1315.5
- 5	- 20.6	105	40.6	215	101.7	1300	704.	2450	1343.
ō	– 17.8	110	43.3	225	107.2	1350	732.	2500	1371.
+ 5	- 15.	115	46.I	250	121.2	1400	760.	2550	1399.
10	- 12.2	120	48.9	300	148.9	1450	788.	2600	1426.5
15	- 9.4	125	51.7	350	176.7	1500	816.	2650	1455.
20	-6.7	130	54.4	400	204.4	1550	844.	2700	1483.
25	- 3.9	135	57.2	450	232.2	1600	872.	2750	1510.
30	- 1.1	140	60.	500	260.	1650	899.	2800	1537.5
32	0	145	62.8	550	287.8	1700	926.	2850	1565.
35	+ 1.7	150	65.6	600	315.6	1750	954.	2900	1593.
40	4.4	155	68.3	650	343.3	1800	982.	2950	1621.
45	7.2	160	71.1	700	371.1	1850	1010.	3000	1648.5
50	10.	165	73.9	750	398.9	1900	1038.	3050	1676.
55	12.8	170	76.7	800	426.7	1950	1065.5	3100	1705.
60	15.6	175	79.4	850	454.4	2000	1003.	3150	1732.
65		180	82.2	900	482.2	2050	1121.	3200	1760.

To convert Fahrenheit into Centigrade: Subtract 32 from Fahrenheit, divide remainder by 9 and multiply by 5.

Example: 212 Fahr.

$$\frac{3^2}{180}$$
 180 ÷ 9 = 20. 20 × 5 = 100.

Ans. 212 Fahr. = 100 Cent.

Centigrade to Fahrenheit: Divide by 5, multiply by 9 and add 32. Example: 260 Cent. \div 5 = 52. $52 \times 9 = 468 + 32 = 500$ Fahr. Ans. 260 Cent. = 500 Fahr.

ALLOYS FOR COINAGE

	Gold	Copper	Silver	Other Constitu- ents	Remarks
Gold coin	91.66 90.0 1.33	8.33 10.0 82.73	15.93	=	British standard. "Latin Union" and American. Roman, Septimus Severus, 265 A.D.
" " Silver coin	40.35 0.1	19.63 7.1 7.5	40.02 92.05	Lead 0.2	Early British B.C. 50. Roman B.C. 31, almost same as British silver coin. British standard.

Composition of Bronzes (Navy Department)

White Metal:	PARTS
Tin	. 7.6
Copper	. 2.3
Zinc	
Antimony	
Lead	. 3.0
Hard Bronze for Piston Rings:	
Tin	. 22.0
Copper	. 78 .0
Bearings — Wearing Surfaces, etc.:	
Copper	. 6
Tin	. I
Zinc	. 1
Naval Brass:	
Copper	. 62.0
Tin	. 1.0
Zinc	. 37.0
Brazing Metal:	
Copper	. 85.0
Zinc	
Antifriction Metal:	Ū
Copper — (best refined)	. 3.7
Banca tin	
Regulus of antimony	
Well fluxed with borax and rosin in mixing.	, ,
Bearing Metal — (Pennsylvania Railroad):	
Copper	. 77.0
Tin	. 8 .o

BEARING METAL

In the Journal of the Franklin Institute G. H. Clamer states that 13 parts antimony and 87 parts lead make an excellent bearing metal, these being exactly the proportions which give a homogeneous structure. For heavier duty tin should be added.

BISMUTH ALLOYS (FUSIBLE METALS)

	Bismuth	Lead	Tin	Cadmium	Melting Point
Newton's alloys Rose's " Darcet's " Wood's " Lipowitz's "	50.0 50.0 50.0 50.0 50.0	31.25 28.10 25.00 24.00 27.00	18.75 24.64 25.00 14.00 13.00	 I2.00 I0.00	C° · 95 100 93 66-71 60

ALLOYS

	Copper	Tin	Lead	Zinc	Nickel	Anti- mony	
Babbitt	8.	92.				4.	Very hard.
Bell metal	76.5	23.5					"Big Ben,"
	74.8	25.2					Westminster.
Brass				27-34			Typical brass.
" wire	70.29	0.17		29.26			
Britannia	1.46	90.62				7.81	Birmingham
		-					sheet.
Bronze	95	4.		I.			British coinage.
	80-00			7.			Heavy bearings.
German silver				20.	20		Nickel varies.
Gun metal	QI	9.					Cannons.
Mannheim gold			}	20-12			
Muntz metal				38-40			Ship sheathing.
Packfong				40.6			Chinese alloy.
Shot metal			99.6	40.9	15.6		Trace of arsenic.
Speculum			99.0	trace			Telescope mir-
specurum	70.24	29.11		liace			ror.
Type metal			70		1	18.	101.
Type metal	2.0	10.	70				Ctomostumina
W/h:41	_	3.2	82			14.	Stereotyping.
White metal	6.	82.				12.	For bearings.

Brass Alloys

Strictly, a brass is a copper-zinc alloy containing one-third zinc and two-thirds copper; whereas a bronze is a copper-tin alloy containing approximately 10 per cent. tin and 90 per cent. copper. The old-style gun metal contained from 90 to 92 per cent. copper and from 8 to 10 per cent. tin. Lead is frequently added to both these classes of alloy to make them machine more easily, and both tin and zinc are commonly used in the same alloy, so that today we have a series of copper-tin-zinc alloys of almost infinite variety. In all cases in the useful alloys of this class, however, there is present more than 50 per cent. copper.

In most of the modern alloys tin is depended upon to give strength and zinc to cheapen the mixture. Some of the old-style gun metals contained as much as 16 per cent. tin and 84 per cent. copper, but such metals were brittle and hard. The common yellow brass employed by plumbers in making ordinary valves and fittings may be considered as composed of approximately 16 pounds copper, 8 pounds zinc, and ½ pound lead. It will be noticed that this consists of approximately one-third zinc and two-thirds copper, with a little lead added to improve the machining qualities. For the making of high-grade casting ingots or new metal should be used in all cases. In making a brass the copper should be melted first and the zinc added, care being taken not to let the temperature rise too high, for if it does the zinc will ignite and burn. The lead is added last and the metal thoroughly stirred.

PROPERTIES OF METALS

Metal	Melting Point	Wt. per Cu. In.	Wt. per Cu. Ft.	Tensile Strength	Specific Gravity	Chemical Symbol
Aluminum Antimony Bismuth Brass, cast Bronze Chromium	1217 1166 518 1692 1692 2750	.0924 .2424 .354 .3029 .319	159.63 418.86 611.76 523.2 550. 429.49	20,000 24,000 36,000	2.56 6.71 9.83 8.393 8.83 6.8	Al. Sb. Bi.
Cobalt Copper Gold Iridium	2714 1981 1945 4172	.307 .322 .6979 .8099	530.6 556. 1206.05 1400.	36,000	8.5 8.9 19.32 22.42	Co. Cu. Au. Ir. Fe.
Iron, cast Iron, wrought Lead Manganese	2700 2920 621 2237	.26 .278 .41 .289	450. 480.13 710. 499.4	16,500 50,000 3,000	7.21 7.7 11.37 8.	Fe. Fe. Pb. Mn. Hg.
Mercury Nickel Platinum Silver	-36 2646 3191 1761	.4909 .3179 .7769 .3805	848.35 549.34 1342.13 657.33	40,000	13.59 8.8 21.5 10.53	Ni. Pt. Ag.
Steel — cast Steel — rolled Tin Tungsten Vanadium	2450 2600 449 5430 3146	.28 .2833 .2634 .69	481.2 489.6 455.08 1192.31 343.34	50,000 65,000 4,600	7.81 7.854 7.29 19.10 5.50	Sn. W. V.
Zinc	786	.245	430.	7,500	6.86	Žn.

SHRINKAGE OF CASTINGS

Aluminum — pure "Nickel Alloy. "Special Alloy Iron, Small Cylinders "Pipes "Girders and Beams "United Cylinders of District	.2031 .1875 .1718 .0625 .125	inch	per 	foot "" ""
" Large Cylinders, Contraction of Diameter at Top " Large Cylinders, Contraction of Diameter	.0625	"	"	"
at Bottom	.083	"	"	"
" Large Cylinders, Contraction of Length	.004	"	"	"
Brass — Thin	.167	"	"	"
" Thick	1.150	"	"	"
Copper	.1875	"	"	"
Bismuth	.1563	"	"	"
Lead	.3125	"	"	"
Zinc	.3125	"	"	"

Aluminum

Can be melted in ordinary plumbago crucibles the same as brass and will not absorb silicon or carbon to injure it unless overheated. Melts at 1217 degrees Fahr. or 625 Cent. Becomes granular and easily broken at about 1000 Fahr.

Shrinkage of pure aluminum	.2031" per foot
Nickel Aluminum Casting Alloy	.1875" " " "
Special Casting Alloy	.1718" " "

The most used alloys have a strength of about 20,000 pounds to square inch at a weight of one third that of brass.

Iron or sand molds can be used and should be poured as cool as

it will run to avoid blowholes.

Burnishing. — Use a bloodstone or steel burnisher, with mixture of melted vaseline and kerosene oil or two tablespoonfuls of ground borax, dissolved in a quart of hot water and a few drops of ammonia added.

Frosting. — Clean with benzine. Dip in strong solution of caustic soda or potash, then in solution of undiluted nitric acid. Wash thoroughly in water and dry in hot sawdust.

Polishing. — Any good metal polish that will not scratch will clean aluminum. One that is recommended is made of

Stearic Acid — One part

Fuller's Earth — One part Rotten Stone — Six parts Grind fine and mix very well.

Castings are cleaned with a brass scratch brush, run at a high speed. Sand blasting is also used both alone and before scratch brushing.

Spinning. — A high speed, about 4000 feet per minute, is best for spinning. This means that for work 5 to 8 inches in diameter, 2800 to 2600 revolutions per minute is good, while for smaller work of 4 inches this would go up to 3200 r.p.m.

Turning. — Use a tool with shearing edge similar to a woodcutting tool as they clear themselves better. Use kerosene or water as a lubricant, or if a bright cut is wanted use benzine. For drawing on a press use vaseline.

Soldering. — See page 92.

U. S. Armory Method of Bluing Steel.—Have work clean and free from grease. Take 10 parts of nitre and 1 part of manganese. in this mixture to from 700 to 800 degrees F. and quench in oil.

STEAM HAMMERS AND DROP FORGING

WHILE it is impossible to accurately rate the capacity of steam hammers with respect to the size of work they should handle, on account of the greatly varying conditions, a few notes from the experience of the Bement works of the Niles-Bement-Pond Company

will be of service.

For making an occasional forging of a given size, a smaller hammer may be used than if we are manufacturing this same piece in large quantities. If we have a 6-inch piece to forge, such as a pinion or a short shaft, a hammer of about 1100 pounds capacity would answer very nicely. But should the general work be as large as this, it would be very much better to use a 1500-pound hammer. If, on the other hand, we wish to forge 6-inch axles economically, it would be necessary to use a 7000- or 8000-pound hammer. The following table will be found convenient for reference for the proper size of hammer to be used on different classes of general blacksmith work, although it will be understood that it is necessary to modify these to suit conditions, as has already been indicated.

Diameter of Stock	Size of Hammer
3½ Inches 4 Inches 4½ Inches 5 Inches 5 Inches	250 to 350 pounds 350 to 600 pounds 600 to 800 pounds 800 to 1000 pounds 1100 to 1500 pounds

Steam hammers are always rated by the weight of the ram, and the attached parts, which include the piston and rod, nothing being added on account of the steam pressure behind the piston. This makes it a little difficult to compare them with plain drop or tilting hammers, which are also rated in the same way.

Steam hammers are usually operated at pressures varying from 75 to 100 pounds of steam per square inch, and may also be operated by compressed air at about the same pressures. It is cheaper, however, in the case of compressed air to use pressures from 60 to 80

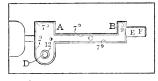
pounds instead of going higher.

In figuring on the boiler capacity for steam hammers, there are several things to be considered, and it depends upon the number of hammers in use and the service required. It will vary from one boiler horse-power for each 100 pounds of falling weight up to three horse-power for the same weight, according to the service expected. In a shop where a number of steam hammers are being used, it is usually safe to count on the lower boiler capacity given, as it is practically safe to say that all of the hammers are never in use at the same time. In a shop with a single hammer, on the other hand, and especially where hard service is expected, it is necessary to allow the larger boiler capacity as there is no reserve to be drawn on, due to part of the hammers being idle, as in the other case.

DRAFT IN DROP FORGING DIES

In sinking dies for drop forging, it is important that the draft at the sides of the impression should be made as little as possible to avoid heavy cuts in the machining operations. It is equally important that the draft be sufficient to allow the forging to be easily withdrawn from the die, else production under the hammer will be hampered. The standard draft (or draw) for most dies is 7 degrees from the perpendicular, but other angles are used in special cases and sometimes two or three different angles of draft are used in the same die at different parts of the impression.

Figure 1 shows the plan and side elevation of a lower die where three angles of draft are advisable. The shoulders A and B are



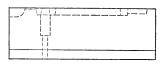


Fig. 1. — Example of Draft in Drop Forging Dies

places where the metal is likely to hug on account of the contraction of the hot metal along the part marked C, which is of comparatively small cross-section and will cool rapidly. These shoulders are given an angle of 9 degrees. The inserted tool-steel plug D, is another place where metal is likely to hug badly. It is usual to give such plugs 12 and even 15 degrees of draft on each side. Moreover, the tendency of plugs to get "jumped up," hammered over and badly heat checked is much reduced if they are given a big draft. The part marked E is semicircular and will draw easily. The end of E at F is a part of a sphere. All other sides of the impression are given 7 degrees draft. If the die is smooth and regular the forging will draw easily.

All impressions are laid out to one-eighth inch to the foot shrink rule. This allows for a shrinkage of about 0.010 inch to the inch. The same shrinkage is allowed in the thickness of forgings. For example: A forging is to be 2 inches thick with half the thickness in each die. The depth of impressions will be 1.010 inch in each die. In laying out the impression on the face of the die, allowance has been made for shrinkage in length, thickness and breadth of forging, and, in addition, for the draft on the sides. In complex dies where there are many different depths and offsets on the face of the die, the die sinker has to keep all these points constantly in mind while laying out or run the risk of spoiling the whole job.

A Table of Draft Dimensions

The allowance for 7-degree draft is easy to remember, being almost exactly $\frac{1}{32}$ inch at the face of the die for each $\frac{1}{4}$ inch of depth. Table 1 has been calculated to give the allowances in thousandths of an

inch on actual depth, as measured with an ordinary depth gage. It is not usual in marking out to work closer than $\frac{1}{6+}$ inch, but the

Table 1.—Allowance in Thousandths of an Inch at Face of Die for Standard Angles of Draft and Various Depths of Impression

Depths in	STANDARD DRAFT ANGLES IN DEGREES					
Inches	5 Degrees	7 Degrees	9 Degrees	12 Degrees		
	Inch	Inch	Inch	Inch		
18	0.011	0.015	0.020	0.027		
į į	0.022	0.031	0.040	0.053		
© 80+ 445 80 9 4 7- 80	0.033	0.046	0.059	0.080		
1/2	0.011	0.061	0.079	0.106		
5 8	0.055	0.077	0.099	0.133		
3 4	0.066	0.092	0.119	0.159		
78	0.077	0.107	0.139	0.186		
1	0.087	0.123	0.158	0.213		
L1/8	0.098	0.138	0.178	0.239		
11	0.100	0.153	0.198	0.266		
I 3	0.120	0.169	0.218	0.292		
1 1/2	0.131	0.184	0.238	0.319		
1 5	0.142	0.200	0.257	0.345		
1 ½ 1 5 5 1 3 4 1 7 5	0.153	0.215	0.277	0.372		
I 7/8	0.164	0.230	0.297	0.399		
2	0.175	0.2.16	0.317	0.425		

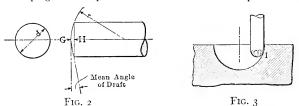
arrangement of table in thousandths allows the nearest $\frac{1}{64}$ inch to be taken. The best plan is to take the allowance for the angle at 1 inch depth as a constant and figure out the allowance for the particular depth wanted from the expression

 $Draft \ allowance = \frac{C \ D}{I} \ .$

Where C = a constant and D = the depth in inches.

MAKING TYPES

A type, shown in Fig. 2, is generally used as a guide for chipping and scraping out the spherical end of the semicircular part E. It is



not usual to make the curve on the end conform to any particular mean angle of draft. Most diesinkers merely turn the end of the type to a curve that looks right to the eye. However, uniformity is desirable in these curves and Table 2 is given as a help in that

Table 2. — Mean Draft of Spherical End of Cylindrical

When Radius of End of Type is	Mean Angle of Draft in Degrees is
$2\frac{1}{2}$ × Diameter of Type $2\frac{1}{4}$ × " " " 2 × " " " " 1 $\frac{1}{4}$ × " " " " 1 $\frac{1}{2}$ × " " "	5 ³ / ₄ 6 ¹ / ₂ 7 ¹ / ₄ 8 ¹ / ₄ 9 ¹ / ₂

direction. It gives the values of the mean angle of draft with various radii expressed in terms of the diameter of the type. It will be noted that if the radius r is made twice the diameter of the type the mean angle of draft is $7\frac{1}{4}$ degrees. This rule is easy to remember and a good one to adopt as standard. A good and easy way to get a close approximation to the required curve is as follows: Turn a cylinder of tool steel to the required diameter. Face the end square, scratch off the distance GH equal to the allowance for draft obtained from Table 1, remembering that the depth is half

the diameter of the type. Turn the end to a curve which is uniform to the eye from the center to the scratched line H. After the type

is hardened it is ready for use.

Semicircular impressions are finished with ball cutters of the correct diameter. When a ball cutter of the correct diameter is not at hand and the job will not warrant making one, the following method may be used. The center



FIG. 4

line of the impression is projected to the end of the die. A semicircle is scribed on the vertical surface of the end. After the impression is roughed out, a smaller ball cutter is placed in the chuck of a diesinking machine and the knee and slides manipulated until the cutter is in proper relation to the semicircle as shown in Fig. 3. A square is used to indicate when the curves of cutter and semicircle are coincident, as at I. The micrometer dials are now set, the lateral slide locked, the knee lowered, the longitudinal slide operated until the cutter is in position over the impression and the ball tool sunk into the die until the micrometer comes to the position set at the semicircle on the end. A longitudinal cut is taken with this setting. cutter is then placed in another lateral position and the operation of setting and cutting repeated. It may be necessary to perform this operation several times, and even then the result will be a series of gutters and ridges instead of a uniform, semicircular depression. This can be readily corrected with the scraper and riffler.

An aid in testing the accuracy of semicircular impressions is shown in Fig. 4. If the semicircle is true, the corners of the square will touch in all positions when the sides are resting on the edges of the impression. If the square rocks on the corner in any position, that spot is high and must be scraped down. This test must be made

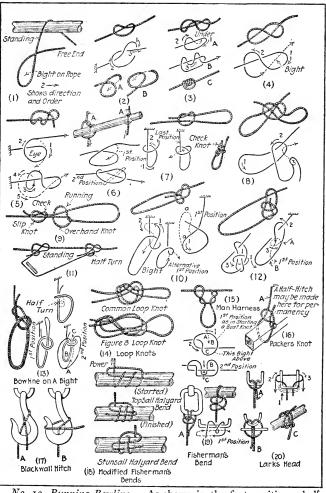
before the "flash" is milled in the die.

KNOTS AND SLINGS FOR HANDLING WORK

THE knots described have been useful in work in out-of-the-way places. No. r indicates the meaning of the terms employed.

- No. 2, Simple or Overhand Knot. The simplest of all knots to tie, and may be used as a stop on a rope. A free end is necessary to make it. If strained, it injures the fiber of the rope more than a figure-8 knot, and it is difficult to unmake and liable to jam.
- No. 3, Double Overhand Knot. Used for the end of a rope when it is required to prevent its going through an eye, as in a pulley block or for the end of a halter rope. Also useful for shortening a rope and may be made with any number of turns: A in the illustration shows the first position; B, the knot finished with two turns; and C, one with four full turns of rope.
- No. 4, Figure-8 Knot (Flemish). May be employed as a stop on a rope; is less injurious to the fiber of the rope, and more easily undone than either the single or double overhand knot. If made with the rope doubled and the bight left long, it becomes a figure-8 hoop knot.
- No. 5, Stevedore Knot. End of the rope is wrapped twice around the standing and then passed through the cye. Useful as a stop on a rope to prevent the end going through an eye, as in a pulley block (see double overhand knot). Also employed instead of sewing the rope end with twine.
- No. 6, Boat Knot (Marline-spike Hitch). Suitable for quickly making a rope ladder, or getting a temporary pull on a rope with a marline-spike. No free ends required to form this knot. Point marked A must always be at the back of the spike or rung of the ladder, away from the direction of the weight or pull.
- No. 7, Slip Knot (Simple Running Knot). The simplest kind of slip knot. It may be used similarly to the packer's knot, but is not so good, as it is liable to pull through and does not bind on the rope.
- No. 8, Tomfool Knot (Double Running Knot). When the loops are drawn taut and the ends tied, this makes a pair of handcuffs which it is almost impossible for the person so secured to undo. It may be used as a barrel sling, half-hitches being put on the ends, and the hook put under the knot itself. The bight marked 3 is passed through the overhand loop as shown by the dotted line.
- No. 9, Flemish Loop. This knot makes a simple loop for light work and may be used in the same way as a bowline, but is not so quickly made; neither is it so secure nor so easily undone. The security depends almost entirely upon the check knot.
- No. 10, Bowline. A generally useful knot when a loop of any sort that will not slip is required, as in a sling for lowering a man, or fastening a bucket to a rope.
- No. 11, Bowline II. A method of attaching the end of one rope to the standing of another. A half turn is put in the standing and the

end of the other rope taken through as if tying an ordinary bowline. This knot is practically a sheet bend.



No. 12, Running Bowline.—As shown in the first position a half turn is made at A (shown dotted) and the end is passed through and to the back of the part marked B. This is a good slip knot and does not tighten on the standing, always remaining open.

No. 13, Bowline on a Bight. — The part marked A is passed behind B and then in the direction of the arrow to C. The bight B is then pulled taut. The two loops of this knot may be used as a man sling, a barrel sling, or as a double man-harness, one loop under each shoulder. When tightened it will not slip.

In case of an injured man, one of the loops can be kept shorter than the other and adjusted under the armpits, the man being

seated in the larger loop.

No. 14, Open-hand Loop Knot and Figure-8 Loop Knot. — The upper loop knot is the one in common use and is adapted principally for small ropes. The lower, or Figure-8 knot, is a better form and may be used on a larger rope as it injures the fiber less than the common form. These knots require a greater length of rope than the bowlines, but may be used in similar ways.

No. 15, Man-harness Knot. — This knot can be tied in a rope with neither end free. The bight A is pulled through under B and over C, and the knot pulled taut. It is useful as allowing a number of men to get a good purchase on a rope for hauling; also to put loops

in the rope to receive hooks at points other than the ends.

No. 16, Packer's Knot. — A modification of a simple slip knot, but has the advantage, when pulled tight, of biting on the standing at A and not easily slipping back. It is particularly useful for cording up rolls of camp bedding, etc. It can be made permanent by an added half hitch on the standing.

No. 17, Blackwall Hitch. \rightarrow A convenient method for returning an empty rope on a hook. With a greasy rope, method B holds better.

No. 18, Modified Fisherman's Bends. — These are given as alternatives for securing ropes to poles or bars, and are adapted to heavy strains.

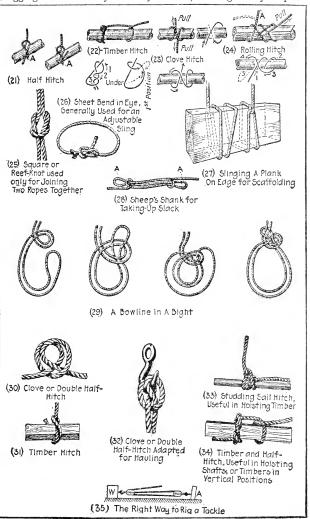
No. 10, Fisherman's Bend. — A better method than the gooseneck or lark's head (Fig. 20) for securing a rope to a chain or link. It is also used to fasten the rope to a bar or the bail of a bucket. Lashing at A is necessary to prevent pulling through. As shown in first position, two turns are taken over the link and the end brought back in front and passed through the turns as shown dotted.

No. 20, Lark's Head. — Useful for fastening a rope to the link of a chain or to a ring in a wall or box. It is not a secure knot unless lashed at A. B shows a toggle inserted to prevent slipping. C is a modified gooseneck on a bar, suitable for securing the end of a rope in scaffolding. The end must be placed at the back and the whole pulled taut.

No. 21, Half Hitch. — A quick and simple way of securing a rope to a timber when no great pull is expected. The rope end is placed under the pole, then back over to the right as shown. The end must always be placed right at the back away from the pull, as shown at A. The right-hand sketch shows the hitch with a slip to facilitate undoing.

No. 22, Timber Hitch. — This is the best and simplest of all timber hitches and may be used for towing or otherwise handling timber, rods, pipes, etc.; also for starting lashings on scaffolding or any kind of pole work. For raising or lowering timber, the half hitch should be placed high above the center of gravity to avoid slanting.

No. 23, Clove Hitch. — This is one of the most useful of all hitches, as it will take a strain in either direction without slackening. It is used for mooring ships heads of derricks for guy lines and all kinds of rigging work. It may be easily undone, or a bight may be put in



instead of one end to use as a slip. When commencing to tie the hitch on a horizontal bar, the rule is over and back below, or the reverse of the procedure in tying a half hitch.

No. 24, Rolling Hitch. — This lashing is used for getting a grip on a large rope with a smaller one. Made in a chain it can be applied to wire ropes and will not slip when the load has been taken up. It is also suitable for hauling on electric cables, or withdrawing diamond drill or other rods. For securing the end A may be brought down and be lashed to the large rope. In making, the end is passed over the spar twice, then returned back as shown at (3), then over behind as at (4), and up and under as at (5).

 $\mathit{No.}$ 25. — A square or reef knot, used only for joining two ropes together.

No. 26. — Sheet bend in an eye, generally used for an adjustable sling.

In supporting a swinging scaffold, it is often advantageous to use light material, while, at the same time, strength is required. A plank on edge is a great deal stiffer than the same plank laid flat, and No. 27 shows how to sling a plank edgewise by a rope so that it will stay. The knot used is a very simple one. A clove hitch is made around the end of the plank; then one of the parts is twisted around the plank until the ends lead as shown in the sketch.

Very often it is desirable to shorten a piece of rope without cutting it. No. 28 shows a sheep's shank which is used for this purpose. The rope is brought back on itself, making two or more bights, and a half hitch is taken around each bight. This knot will not slip, and will nearly fall apart of its own accord if the strain is released, so that when there is a liability of this happening, it is well to pass a piece of wood through the loop Λ at each end and pull the rope tight on them.

One of the handiest knots to know is a bowline. The bowline will not slip, and is easy to untie. It can also be tied in the bight of a rope, and is then called a "bowline in a bight." The steps required to tie it are shown at No. 29. It is particularly handy when it is necessary to hitch an auxiliary tackle on a fall to get additional purchase for a heavy lift. This knot has all the good points of the

simple bowline.

In using a block and fall for pulling things, there is a right way and a wrong way of doing it. No. 35 shows the right way, IV being the weight to be moved. If A were the weight and W the post, the blocks being left as shown, then it would be wrong. The advantage of the right way of doing it is that the leverage due to one additional part of rope in the tackle is gained! thus a three-part fall, rigged in the right way, is as good as a four-part fall rigged in the wrong way, and has the additional advantage that there is one less sheave with its friction. In lifting a heavy weight, it is sometimes desirable to put a tackle on the fall to gain additional leverage; the common practice in a case of this kind is to hitch the auxiliary tackle to a "dead man." The right way is to hitch this tackle to the piece to be lifted alongside the main tackle, which adds considerably to the

leverage, being equivalent to one more part to the main fall besides the gain by the use of the auxiliary fall.

No. 30. — Clove or double half hitch.

No. 31. — Timber hitch.

No. 32. — Clove or double half hitch as used for hauling.

No. 33. - Studding sail hitch as used in hoisting timber.

No. 34. — Timber and half hitch. Useful in hoisting shafting or timber in a vertical position.

SAFE LOADS FOR EYE-BOLTS AND FOR ROPES AND CHAINS

TABLE I. - SAFE LOADS FOR EYE-BOLTS

	A	Inches B	С	Safe Load, Lb.
Drop-forged steel	1.2 9 0 1.5 is 2 1.4 7 is 1.4 1.4 1.4 1.4 1.4 1.4 1.4 1.4 1.4 1.4	$\begin{array}{c} \mathbf{I} \begin{array}{c} \mathbf{I} \begin{array}{c} \mathbf{I} \\$	7 1 7 5 5 8 3 4 7 10 5 5 9 10 7 10 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	1,100 1,500 1,800 2,800 3,900 5,100 8,400 12,200 16,500
D.B G. iron E.L., 28,000 lb. per sq. in., welded	2 I 34 I 34 2 2 24	318 3 4 5 6	I 55 8 I 144 1 12 3 14 2 2	21,800 10,000 11,000 14,000 16,000

TABLE 2. - SAFE LOADS ON ROPES AND CHAINS

Manila Rope, Safe Load in Tons				Wire Cable, Safe Load in Tons			Chai	ns, Safe	Load in	Tons	
Diam. of Rope, In.	Single Rope	Two Part	Four Part	Diam. of Cable, In.	Single Rope	Two Part	Four Part	Diam. of Chain, In.	Single Chain	Two Part	Four Part
1	1601 433 F12334 I I 2 2 3 4 4	141234 1 12 2 24 5 62 8	1 2 1 2 3 4 6 8 H I I 3	1(25)(83)47 8 I I 144 2334 I I 24	1 134 222 334 4 6 10 13 16	2 3 \frac{1}{4} \frac{1}{2} \frac{2}{1} \frac{1}{2} \f	3 1/2 1 6 1/2 9 1 2 1 6 2 4 3 6 4 8 6 0	1/43)2-1/245/003/47/6 1/80-1/4-1/2	1 2 3 5 6 8 11 13 18	1 1 1 2 2 3 3 2 3 2	1 ½ 3 6 9 15 18 24 33 39 54

GENERAL REFERENCE TABLES

COMMON WEIGHTS AND MEASURES

LINEAR OR MEASURE OF LENGTH

12 inches = 1 foot. 3 feet = 1 yard. $5\frac{1}{2}$ yards = 1 rod. 40 rods = furlong. 8 furlongs = 1 mile.

Equivalent Measures

Inches	3	Feet		Yards		Rods	$\mathbf{F}_{\mathbf{i}}$	urlongs	Mile
36	===	3	=	I					
198	=	16.5	=	5.5	=	I			
7920	===	660	==	220	===	40	=	I	
63,360	= 5	5280	= 1	760	=	320	=	8 =	1

Souare Measure

144 square inches = 1 sq. foot. 30¼ square yards = 1 sq. rod.
9 square feet = 1 sq. yd. 160 square rods = 1 acre.
640 acres = 1 sq. mile.

EQUIVALENT MEASURE

Sq. Mi. A. Sq. Rd. Sq. Yd. Sq. Ft. Sq. In. 1 = 640 = 102,400 = 3,097,600 = 27,878,400 = 4,014,489,600

CUBIC MEASURE

1728 cubic inches = 1 cubic foot. 128 cubic feet = 1 cord. 27 cubic feet = 1 cubic yard. 24 $\frac{3}{4}$ cubic feet = 1 perch. 1 cu. yd. = 27 cu. ft. = 46,656 cu. in.

Weight - Avoirdupois

437.5 grains = 1 ounce. 100 pounds = 1 hundred weight.
16 ounces = 1 pound. 2000 pounds = 1 ton.
2240 pounds = 1 long ton.
1 ton = 20 cwt. = 2000 lbs. = 32,000 oz. = 14,000,000 gr.

Weight = Troy

24 grains = 1 pennyweight. 20 pwt. = 1 ounce.
12 ounces = 1 pound.
1 lb. = 12 oz. = 240 pwt. = 5760 gr.

(74)

Dry Measure

2 pints = 1 quart. 8 quarts = 1 peck. 4 pecks = 1 bushel1 bu. = 4 pk. = 32 qt. = 64 pt.U. S. bushel = 2150.42 cu. in. British = 2218.19 cu. in.

LIQUID MEASURE

4 gills = 1 pint. 4 quarts = 1 gallon. 2 pints = 1 quart. 31½ gallons = 1 barrel. 2 barrels or 63 gals. = 1 hogshead. 1 hhd. = 2 bbl. = 63 gals. = 252 qt. = 504 pt. = 2016 gi.

The U. S. gallon contains 231 cu. in. = .134 cu. ft.

One cubic foot = 7.481 gallons.

One cubic foot weighs 62.425 lbs. at 39.2 deg. Fahr. One gallon weighs 8.345 lbs.

For rough calculations I cu. ft. is called 7¹/₄ gallons and I gallon as $8\frac{1}{3}$ lbs.

Angles or Arcs

60 seconds = 1 minute. 90 degrees = 1 rt. angle or quadrant. 60 minutes = 1 degree. 360 degrees = 1 circle.
1 circle = 360° = 21,600′ = 1,296,000″. I minute of arc on the earth's surface is I nautical mile = 1.15 times a land mile or 6080 feet.

Weight of a Cubic Foot of Substances	
Avera Names of Substances Weigi Lbs.	ht
Anthracite, solid, of Pennsylvania	3
"broken, loose 5.	
" moderately shaken 55	
" heaped bushel, loose	
Ash, American white, dry	8
Asphaltum 8	7
Brass (Copper and Zinc), cast	4
" rolled 522	4
Brick, best pressed	ò
" common hard 12	5
" soft, inferior 100	Ö
Brickwork, pressed brick 140	0
" ordinary 11:	2
Cement, hydraulic, ground, loose, American, Rosendale	6
" " " Louisville 50	0
" " English, Portland	0
Cherry, dry4	2
Chestnut, dry 4	I
Coal, bituminous, solid	
" hroken loose	'n

broken, loose heaped bushel, loose

WEIGHT OF A CUBIC FOOT OF SUBSTANCES — Continued

NAMES OF SUBSTANCES	Weight Lbs.
Coke, loose, of good coal	. 27
Copper, cast	. 542
" rolled	. 548
Earth common loam dry loose	. 76
Earth, common loam, dry, loose """ moderately rammed	. 95
Ebony, dry	. 93
Elm, dry	
Flint	
Glass, common window	
Gneiss, common	
Gold, cast, pure, or 24 carat	
" pure, hammered	
Granite	
Gravel, about the same as sand, which see.	. 1/0
Hemlock, dry	. 25
Hickory, dry	
Hornblende, black	
Ice	
Iron, cast	450
" wrought, pure	
" average	
Ivory	
Lead	
Lignum Vitæ, dry	83
Lignum Vitæ, dry Lime, quick, ground, loose, or in small lumps	53
" " thoroughly shaken	75
Limestones and Marbles	75 168
Limestones and Marbles	96
Mahogany, Spanish, dry	53
"Honduras, dry	35
Maple, dry	
Marbles, see Limestones.	-
Masonry, of granite or limestone, well dressed	165
" sandstone, well dressed	. 144
Mercury, at 32° Fahrenheit	849
Mica	
Mortar, hardened	103
Mud, dry, close 80 to	110
" wet, fluid, maximum	120
Oak, live, dry	59
" white, dry	
" other kinds	
Petroleum	
Pine, white, dry	
" yellow, Northern	. 34
" Southern	45

WEIGHT OF A CUBIC FOOT OF SUBSTANCES - Continued

Names of Substances	Weight Lbs.
Platinum	1342
Quartz, common, pure	165
Rosin	69
Salt, coarse, Syracuse, N. Y.	45
" Liverpool, fine, for table use	49
Sand, of pure quartz, dry, loose	106
" well shaken 99 to	117
" perfectly wet 120 to	140
Sandstones, fit for building	151
Shales, red or black	162
Silver	655
Slate	175
Snow, freshly fallen 5 to	12
" moistened and compacted by rain 15 to	50
Spruce, dry	25
Steel	490
Sulphur	125
Sycamore, dry	37
Tar	62
Tin, cast	459
Turf or Peat, dry, unpressed 20 to	30
Walnut, black, dry	38
Water, pure rain or distilled, at 60° Fahrenheit	$62\frac{1}{3}$
" sea	64
Wax, bees	60.5
Zinc or Spelter	437
Green timbers usually weigh from one-fifth to one-half more th	an dry.

WATER CONVERSION FACTORS

U. S. gallons U. S. gallons U. S. gallons U. S. gallons U. S. gallons	× 8.33 × 0.13368 × 231 × 0.83 × 3.78	= pounds. = cubic feet. = cubic inches. = English gallons = liters.
English gallons (Imperial) English gallons (Imperial) English gallons (Imperial) English gallons (Imperial) English gallons (Imperial)	X 10 X 0.16 X 277.274 X 1.2 X 4.537	= pounds. = cubic feet. = cubic inches. = U. S. gallons. = liters.
Cubic inches of water (39.1°) Cubic inches of water (39.1°) Cubic inches of water (39.1°) Cubic inches of water (39.1°)	X 0.004329 X 0.003607	 pounds. U. S. gallons. English gallons. ounces.
Cubic feet (of water) (39.1°) Cubic feet (of water) (39.1°) Cubic feet (of water) (39.1°) Cubic feet (of water) (39.1°)	X 7.48 X 6.232	= pounds. = U. S. gallons. = English gallons. = tons.
Pounds of water Pounds of water Pounds of water Pounds of water	X 27.72 X 0.01602 X 0.12 X 0.10	= cubic inches. = cubic feet. = U. S. gallons. = English gallons.

CONVENIENT MULTIPLIERS

Inches Inches Inches	× 0.08333 = feet. × 0.02778 = yards. × 0.00001578 = miles.	Sq. inches Sq. inches Cu. inches Cu. inches	X 0.00695 X 0.0007716 X 0.00058 X 0.0000214	= Sq. feet. = Sq. yards. = Cu. feet. = Cu. yards.
Feet Feet	× 0.3334 = yards. × 0.00019 = miles.	Sq. feet Sq. feet	× 144 × 0.1112	= Sq. inches. = Sq. yards.
Yards Yards Yards	$\begin{array}{ccc} \times & 36 & = \text{ inches.} \\ \times & 3 & = \text{ feet.} \\ \times & 0.0005681 & = \text{ miles.} \end{array}$	Cu. feet Cu. feet Sq. yards	× 1728 × 0.03704 × 1296	Cu. inches.Cu. yards.Sq. inches.
Miles Miles Miles	\times 63360 = inches. \times 5280 = feet. \times 1760 = yards.	Sq. yards Cu. yards Cu. yards		= Sq. feet. = Cu. inches. = Cu. feet.
	\times 0.0625 = pounds. \times 0.00003125 = tons. \times 16 = ounces.	Avoir. lbs. Avoir. tons Avoir. tons		tons.ounces.pounds.

THE METRIC SYSTEM

The Metric System is based on the Meter which was designed to be one ten-millionth (1505,050) part of the earth's meridian quadrant, through Dunkirk and Formentera. Later investigations, however, have shown that the Meter exceeds one ten-millionth part by almost one part in 6400. The value of the Meter, as authorized by the U. S. Government, is 39.37 inches. The Metric system was legalized by the U. S. Government in 1866.

The three principal units are the Meter, the unit of length, the liter, the unit of capacity, and the gram, the unit of weight. Multiples of these are obtained by prefixing the Greek words: deka (10), hekio (100), and kilo (1000). Divisions are obtained by prefixing the Latin words: $\det(\frac{1}{10})$, centi $(\frac{1}{10})$, and milli $(\frac{1}{100})$. Abbreviations of the multiples begin with a capital letter, and of the divisions with a small

letter, as in the following tables:

Measures of Length

10 millimeters (mm)	ı centimetercm.
10 centimeters	
10 decimeters	
10 meters	ı dekameter
10 dekameters	ı hektometer
10 hektometers =	ı kilometerKm.

MEASURES OF SURFACE (NOT LAND)

100 square millimeters (mm²) = 1 square centimeter	
100 square centimeters = 1 square decimeter	
100 square decimeters = 1 square meter	m².

MEASURES OF VOLUME

1000 cubic millimeters (mm ³)	1	cubic centimeter	cm3.
roog cubic centimeters=	Ι	cubic decimeter	dm³.
1000 cubic decimeters	I	cubic meter	

MEASURES OF CAPACITY

10 milliliters (ml) = 1 centiliter
10 Centiliters
ro deciliters = 1 liter
ro liters = 1 dekaliter Dl.
10 dekaliters
10 hektoliters
Norm — The liter is equal to the volume occupied by a cubic decimeter.

Measures of Weight

10 milligrams (mg) = 1 centigramcg.
10 centigrams = 1 decigramdg.
10 decigramsg.
10 grams = 1 dekagram Dg.
to dekagrams = 1 hektogramHg.
10 hektograms = 1 kilogramKg.
1000 kilograms = 1 tonT.

Note. — The gram is the weight of one cubic centimeter of pure distilled water at a temperature of 39.2° F., the kilogram is the weight of r liter of water; the ton is the weight of r cubic meter of water.

METRIC AND ENGLISH CONVERSION TABLE

MEASURES OF LENGTH

```
 \begin{array}{ll} \text{I meter} & \left\{ \begin{array}{l} 30.37 \text{ inches.} \\ 3.28083 \text{ feet.} \\ 1.0936 \text{ yds.} \end{array} \right. \\ \text{I centimeter} & = .3037 \text{ inch.} \\ \text{I millimeter} & \left\{ \begin{array}{l} 0.3037 \text{ inch.} \\ 25 \text{ inch nearly.} \end{array} \right. \\ \text{I kilometer} & = 0.62137 \text{ mile.} \end{array} \right. \\ \end{array}
```

MEASURES OF SURFACE

ı square meter = { 10.764 square feet. ı.196 square yds. ı square centimeter = .155 sq. in. ı square millimeter = .00155 sq. in.	I square yard = .836 square meter. I square foot = .0020 square meter. I square in. = { 6.452 sq. centimeters:
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MEASURES OF VOLUME AND CAPACITY

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The above the first state of the continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous continuous
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MEASURES OF WEIGHT

```
I gram = 15.432 grains.
I kilogram = 2.2046 pounds.
I metric ton = 

| 1 grain = .0648 grams. |
| 1 grain = .0648 grams. |
| 2.046 pounds. |
| 2.046 pounds. |
| 2.046 pounds. |
| 2.046 pounds. |
| 3 grain = .0648 grams. |
| 4 grain = .0648 grams. |
| 5 grain = .0648 grams. |
| 6 grain = .0648 grams. |
| 1 grain = .0648 grams. |
| 1 grain = .0648 grams. |
| 1 grain = .0648 grams. |
| 2 grain = .0648 grams. |
| 1 grain = .0648 grams. |
| 2 grain = .0648 grams. |
| 3 grain = .0648 grams. |
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| 2 grain = .0648 grams. |
| 3 grain = .0648 grams. |
| 4 grain = .0648 grams. |
| 5 grain = .0648 grams. |
| 6 grain = .0648 grams. |
| 7 grain = .0648 grams. |
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| 1 grain = .0648 grams. |
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| 1 grain = .0648 grams. |
| 1 grain = .0648 grams. |
| 1 grain = .0648 grams. |
| 1 grain = .06
```

Miscellaneous Conversion Factors

- 1 kilogram per meter = .6720 pounds per foot.
- 1 gram per square millimeter = 1.422 pounds per square inch.
- 1 kilogram per square meter = 0.2084 pounds per square foot.

- I kilogram per square meter = 0.2084 pounds per square 1001.
 I kilogram per cubic meter = .0624 pounds per cubic foot.
 I degree centigrade = 1.8 degrees Fahrenheit.
 I pound per foot = 1.488 kilograms per meter.
 I pound per square foot = 4.882 kilograms per square meter.
 I pound per cubic foot = 16.02 kilograms per cubic meter.
 I degree Fahrenheit = .5556 degrees centigrade.
 I Calorie (French Thermal Unit) = 3.968 B. T. U. (British Thermal Unit).
 - 1 Horse Power = { 33,000 foot pounds per minute. 746 Watts.

 - 1 Kilowatt = $\begin{cases} 1000 \text{ Watts.} \\ 1.34 \text{ Horse Power.} \end{cases}$ 44240 foot pounds per minute.

DECIMAL EQUIVALENTS OF FRACTIONS OF MILLIMETERS. (AD-VANCING BY $\frac{1}{100}$ MM.)

mm. Inches	mm. Inches	mm. Inches	mm. Inches
100 = .00039 100 = .0018 100 = .00157 100 = .00157 100 = .00276 100 = .00276 100 = .00276 100 = .00315 100 = .00315 100 = .00354 100 = .00394 100 = .00472 100 = .00472 100 = .00551 100 = .00551 100 = .00591 100 = .00591 100 = .00630 100 = .00748 100 = .00748 100 = .00748 100 = .00748 100 = .00787 100 = .00787 100 = .00787 100 = .00826	100 0.0024 100 0.0024 100 0.0003 100	1	760 = .02992 770 = .03032 770 = .03032 770 = .03071 700 = .03110 800 = .03150 100 = .03189 100 = .03228 800 = .03228 100 = .03228 100 = .03386 100 = .03386 100 = .03386 100 = .03425 100 = .03425 100 = .03465 100 = .03584 100 = .03584 100 = .03584 100 = .03584 100 = .03584 100 = .03584 100 = .03584 100 = .03584 100 = .03584 100 = .03584 100 = .03584 100 = .03584 100 = .03584 100 = .03584 100 = .03584
$\begin{array}{r} \frac{23}{100} = .00906 \\ \frac{24}{100} = .00945 \\ \frac{25}{100} = .00984 \end{array}$	$ \begin{array}{c} \frac{48}{100} = .01890 \\ \frac{49}{100} = .01929 \\ \frac{50}{100} = .01969 \end{array} $	$ \begin{array}{c} \frac{73}{100} = .02874 \\ \frac{74}{100} = .02913 \\ \frac{75}{100} = .02953 \end{array} $	$\frac{98}{100} = .03858$ $\frac{99}{100} = .03898$ $I = .03937$

Decimal Equivalents of Millimeters and Fractions of Millimeters. (Advancing by $\frac{1}{5}\frac{1}{0}$ mm. and 1 mm.)

				5(/
mm.	Inches	mm.	Inches	mm.	Inches	mm.	Inches
$\frac{\frac{2}{50}}{\frac{3}{50}} = \frac{4}{50} = \frac{4}{50}$	= .00079 = .00157 = .00236 = .00315	450 = 450 = 430 = 440 = 450	.03150 .03228 .03307 .03386 .03465	32 = 33 = 34 = 35 =	1.22047 1.25984 1.29921 1.33858 1.37795	72 = 73 = 74 =	= 2.79527 = 2.83464 = 2.87401 = 2.91338 = 2.95275
$ \begin{array}{c} $: .00394 : .00472 : .00551 : .00630 : .00709	455 = 45607 = 4580 = 4500 = 45	.03543 .03622 .03701 .03780 .03858	37 = 38 = 39 = 40 =	1.41732 1.45669 1.49606 1.53543 1.57480	77 = 78 = 79 =	= 2.99212 = 3.03149 = 3.07086 = 3.11023 = 3.14960
$\frac{1}{50} = \frac{1}{50} $.00787 .00866 .00945 .01024	1 = 2 = 3 = 4 = 5 = 5	.03937 .07874 .11811 .15748 .19685	42 = 43 = 44 =	1.61417 1.65354 1.69291 1.73228 1.77165	82 = 83 = 84 =	3.18897 3.22834 3.26771 3.30708 3.34645
150 150 150 150 150 150 150	.01181 .01260 .01339 .01417 .01496	6 = 7 = 8 = 9 = 10 =	.23622 .27559 .31496 .35433 .39370	47 = 48 = 49 =	1.81102 1.85039 1.88976 1.92913 1.96850	87 = 88 = 89 =	3.38582 3.42519 3.46456 3.50393 3.54330
245 245 245 245 245 245 245 245 245 245	.01575 .01654 .01732 .01811 .01890	11 = 12 = 13 = 14 = 15 =	.43307 .47244 .51181 .55118 .59055	52 = 53 = 54 =	2.00787 2.04724 2.08661 2.12598 2.16535	92 = 93 = 94 =	3.58267 3.62204 3.66141 3.70078 3.74015
= = = = = = = = = = = = = = = = = = =	.01969 .02047 .02126 .02205 .02283	16 = 17 = 18 = 19 = 20 =	.62992 .66929 .70866 .74803 .78740	57 = 58 = 59 =	2.20472 2.24409 2.28346 2.32283 2.36220	97 = 98 = 99 =	3.77952 3.81889 3.85826 3.89763 3.93700
= = = = = = = = = = = = = = = = = = =	.02362 .02441 .02520 .02598 .02677	21 = 22 = 23 = 24 = 25 =	.82677 .86614 .90551 .94488 .98425	62 = 63 = 64 =	2.40157 2.44094 2.48031 2.51968 2.55905	0.03937 in.	3.937 " 39.37 "
360 = 3507 = 380 = 360 =	.02756 .02835 .02913 .02992 .03071	$\begin{vmatrix} 27 & = 1 \\ 28 & = 1 \\ 29 & = 1 \end{vmatrix}$	1.02362 1.06299 2.10236 2.14173 2.18110	67 = 68 = 69 =	2.59842 2.63779 2.67716 2.71653 2.75590	1 mm.= 0.	mi iii

EQUIVALENTS OF ENGLISH INCHES IN MILLIMETERS (39.37 inches = 1 meter)

							_										_			_	_			
Inch		Н	01	٠٠.	4	ιC.	9	7		-	OI	I	12	13	14	15	91	17	81	119	20	21	22	23
100	23.8	49.2	74.6	100.0	125.4	150.8	176.2	201.6	227.0	252.4	277.8	303.2	328.6	354.0	379.4	404.8	430.2	455.6	481.0	506.4	531.8	557.2	582.6	608.0
p- s0	22.2	47.6	73.0	98.4	123.8	149.2	174.6	200.0	225.4	250.8	276.2	301.6	327.0	352.4	377.8	403.2	428.6	454.0	479.4	504.8	530.2	555.6	581,0	606.4
133	20.6	46.0	71.4	8.96	122.2	147.6	173.0	198.4	223.8	240.2	274.6	300.0	325.4	350.8	376.2	9.104	427.0	452.4	477.8	503.2	528.6	554.0	579.4	604.8
614	1.61	44.4	8.69	95.2	120.7	146.1	171.5	6.961	222.3	247.7	273.1	298.4	323.8	349.2	374.6	400.0	425.4	450.8	476.2	501.6	527.0	552.4	577.8	603.2
1,1	17.5	42.0	68.3	93.7	1.911	144.5	169.9	195.3	220.7	246.I	271.5	296.9	322.3	347.7	373.1	398.5	423.9	449.3	474.7	500.I	525.5	550.0	576.3	601.7
rajos	15.9	41.3	2.99	92.1	117.5	142.0	168.3	193.7	219.1	244.5	569.6	295.3	320.7	346.1	371.5	396.9	422.3	447.7	473.1	498.5	523.9	549.3	574.7	1.009
Pg I g	14.3	39.7	65.1	90.5	6.511	141.3	1.991	1.261	217.5	242.0	268.3	293.7	319.1	344.5	369.9	395.3	420.7	446.I	471.5	6.964	522.3	547.7	573.1	508.5
-40	12.7	38.1	63.5	6.88	114.3	139.7	165.1	190.5	215.9	241.3	266.7	292.I	317.5	342.9	368.3	393.7	419.1	444.5	469.9	495.3	520.7	546.I	571.5	206.0
7. I.6	11.1	36.5	6.19	87.3	112.7	138.1	163.5	6.881	214.3	239.7	265.1	290.5	315.9	341.3	366.7	392.1	417.5	442.9	468.3	493.7	519.1	544.5	569.9	505.3
mixe	9.5	34.9	60.3	85.7	III.I	136.5	6.191	187.3	212.7	238.I	263.5	288.9	314.3	339.7	365.1	390.5	415.9	441.3	466.7	492.I	517.5	542.9	568.3	503.7
16	7.9	33.3	58.7	84.1	109.5	134.9	160.3	185.7	211.I	236.5	261.9	287.3	312.7	,338.1	363.5	388.9	414.3	439.7	465.1	490.5	515.9	541.3	566.7	502.I
H4	6.4	31.7	57.1	82.5	108.0	133.4	158.8	184.2	9.602	235.0	260.4	285.7	311.1	336.5	361.9	387.3	412.7	438.I	463.5	488.9	514.3	539.7	565.1	500.5
1.8	8.4	30.2	52.6	81.0	106.4	131.8	157.2	182.6	208.0	233.4	258.8	284.2	300.6	335.0	360.4	385.8	411.2	436.6	462.0	487.4	512.8	538.2	563.6	589.0
-4xc	3.2	28.6	54.0	79.4	104.8	130 2	155.6	0.181	206.4	231.8	257.2	282.6	308.0	333.4	358.8	384.2	9.604	435.0	400.4	485.8	511.2	536.6	562.0	587.4
21	9.1	27.0	52.4	27.8	103.2	128.6	154.0	179.4	204.8	230.2	255.6	281.0	306.4	331.8	357.2	382.6	408.0	433.4	458.8	484.2	200.6	535.0	500.4	585.8
0	0.0	25.4	50.8	76.2	9.101	127.0	152.4	177.8	203.2	228.6	254.0	279.4	304.8	330.2	355.6	381.0	406.4	431.8	457.2	482.6	508.0	533.4	558.8	584.2
Tuch	0	н	61	3	4	rv.	9	7	×	6	OI	II	12	13	14	15	91	ĭ	20	61	9	21	22	23

DECIMAL EQUIVALENTS OF FRACTIONS OF AN INCH. (ADVANCING BY STHS, 16THS, 32NDS AND 64THS.)

8ths 32nds $ \begin{array}{cccccccccccccccccccccccccccccccccc$	64ths 64 = .015625 61 = .046875 61 = .078125 61 = .109375 61 = .174625 61 = .171875 61 = .203125	64ths $ \begin{array}{cccccccccccccccccccccccccccccccccc$
16ths. $\frac{\frac{15}{3}}{\frac{5}{3}} = .46875$	$\begin{bmatrix} \frac{3}{64} = .046875 \\ \frac{5}{64} = .078125 \\ \frac{7}{64} = .109375 \\ \frac{9}{64} = .140625 \\ \frac{1}{64} = .171875 \end{bmatrix}$	$\begin{vmatrix} \frac{35}{64} = .546875 \\ \frac{37}{64} = .578125 \\ \frac{86}{1} = .600375 \end{vmatrix}$
16 .0025 32 .55125	$\frac{\frac{15}{64}}{\frac{17}{64}} = .234375$ $\frac{17}{64} = .265625$	$\frac{4}{64} = .734375$
$ \begin{array}{cccccccccccccccccccccccccccccccccccc$	$\begin{array}{cccccccccccccccccccccccccccccccccccc$	$\begin{array}{cccccccccccccccccccccccccccccccccccc$

Decimal Equivalents of Fractions of an Inch. (Advancing by $64 \mathrm{Ths.}$)

$\begin{array}{l} \frac{1}{64} = .015625 \\ \frac{1}{32} = .03125 \\ \frac{3}{64} = .046875 \\ \frac{1}{16} = .0625 \end{array}$	$\begin{array}{l} \frac{17}{64} = .265625 \\ \frac{9}{32} = .28125 \\ \frac{19}{64} = .296875 \\ \frac{5}{16} = .3125 \end{array}$	$\begin{array}{c} \frac{3}{6}\frac{3}{6} = .515625\\ \frac{1}{6}\frac{1}{2} = .53125\\ \frac{3}{6}\frac{1}{4} = .546875\\ \frac{9}{16} = .5625 \end{array}$	$\begin{array}{c} \frac{49}{664} = .765625 \\ \frac{2}{664} = .78125 \\ \frac{2}{364} = .796875 \\ \frac{1}{16} = .8125 \end{array}$
$\begin{array}{c} \frac{5}{64} = .078125 \\ \frac{8}{32} = .09375 \\ \frac{7}{64} = .109375 \\ \frac{1}{8} = .125 \end{array}$	$\begin{array}{c} \frac{2\frac{1}{4}}{6\frac{4}{4}} = .328125 \\ \frac{1}{3}\frac{2}{2} = .34375 \\ \frac{2}{6}\frac{2}{4} = .359375 \\ \frac{3}{8} = .375 \end{array}$	$\begin{array}{c} \frac{37}{64} = .578125 \\ \frac{19}{32} = .59375 \\ \frac{39}{64} = .609375 \\ \frac{5}{8} = .625 \end{array}$	$\begin{array}{cccccccccccccccccccccccccccccccccccc$
$\frac{9}{64} = .140625$ $\frac{5}{32} = .15625$ $\frac{11}{64} = .171875$ $\frac{3}{16} = .1875$	$\begin{array}{cccccccccccccccccccccccccccccccccccc$	$\begin{array}{c} \frac{41}{64} = .640625 \\ \frac{21}{332} = .65625 \\ \frac{43}{64} = .671875 \\ \frac{11}{16} = .6875 \end{array}$	$\begin{array}{c} \frac{57}{664} = .890625 \\ \frac{232}{322} = .90625 \\ \frac{156}{644} = .921875 \\ \frac{15}{16} = .9375 \end{array}$
$ \frac{\frac{13}{64}}{\frac{3}{4}} = .203125 $ $ \frac{7}{32} = .21875 $ $ \frac{5}{64} = .234375 $ $ \frac{1}{4} = .25 $	$\begin{array}{c} \frac{29}{64} = .453125 \\ \frac{15}{32} = .46875 \\ \frac{31}{64} = .484375 \\ \frac{1}{2} = .50 \end{array}$	$\begin{array}{c} \frac{45}{64} = .703125 \\ \frac{23}{32} = .71875 \\ \frac{47}{64} = .734375 \\ \frac{3}{4} = .75 \end{array}$	$\begin{array}{c} \frac{61}{684} = .953125 \\ \frac{81}{332} = .96875 \\ \frac{63}{64} = .984375 \end{array}$

Decimal Equivalents of Fractions below $\frac{1}{2}$ "

Decimal			Fra	ction	al P	arts o	f an	Inch			Decimal
Equivalents	6	7	8	12	14	16	24	28	32	64	Equivalents
.015625 .03125 .035714 .041667							1	1	I	I 2	.015625 .03125 .035714 .041667
.046875 .0625 .071429 .078125					I	1		2	2	3 4	.046875 .0625 .071429
.083333 .09375 .107143 .109375				1			2	3	3	6	.083333 .09375 .107143 .109375
.125 .140625 .142857 .15625		ı	I		2	2	3	4	5	9	.125 .140625 .142857 .15625
.166666 .171875 .178571 .1875	1			2		3	4	5	6	11	.166666 .171875 .178571 .1875
.203125 .208333 .214286 .21875					3		5	6	7	13	.203125 .208333 .214286 .21875
.234375 .25 .265625 .28125			2	3		4	6	7	8	15 16 17 18	.234375 .25 .265625 .28125
.285714 .291666 .296875		2	•		4	5	7	8	10	19	.285714 .291666 .296875
.321429 .328125 .333333 .34375	2			4			8	9	11	21	.321429 .328125 .333333 .34375
·357143 ·359375 ·375 ·390625			3		5	6	9	10	12	23 24 25	•357143 •359375 •375 •390625
.392857 .40625 .41666 .421875				5			10	11	13	26	.392857 .40625 .41666
.428571 .4375 .453125		3			6	7		12	14	27 28 29	•428571 •4375 •453125
•458333 •464286 •46875 •484375							11	13	15	30 31	•458333 •464286 •46875 •484375
. 5	3		4	6	7	8.	12	14	16	32	·5

Decimal Equivalents of Fractions between $\frac{1}{2}''$ and $\mathbf{1}''$

Decimal			Fra	ction	al Pa	rts of	an I	nch			Decimal
Equivalents	6	7	8	12	14	16	24	28	32	64	Equivalents
.515625 .53125 .535714 .541666				•			13	15	17	33 34	.515625 .53125 .535714 .541666
.546875 .5625 .571429		4			8	9		16	18	35 36	.546875 .5625 .571429 .578125
.578125 .583333 .59375 .607143				7			14	17	19	37 38	.5/8125 .583333 .59375 .607143
.609375 .625 .640625			5			10	15		20	39 40 41	.609375 .625 .640625
.642867 .65625 .666666	4			8	9		16	18	2 I	42	.642867 .65625 .666666
.671875 .678571 .6875						11		19	22	44	.671875 .678571 .6875 .703125
.703125 .708333 .714286 .71875		5			10		17	20	23	45	.703125 .708333 .714286 .71875
.734375 .75 .765625			6	9		12	18	2 I	24	47 48 49	•734375 •75 •765625
.78125 .785714 .791666					11		19	22	25	50	.78125 .785714 791666
.796875 .8125 .821429						13		23	26	51 52	.796875 .8125 .821429
.828125 .833333 .84375 .857143	5	6		10	12		20	24	27	53 54	.828125 .833333 .84375 .857143
.859375 .875 .890625		U	7		12	14	21	-4	28	55 56 57	.859375 .875 .800625
.892857 .90625 .916666				11			22	25	29	58	.892857 .90625 .916666
.921875 .928571 .9375					13	15		26	30	59 60	.921875 .928571 .9375
.953125 .958333 .964286 .96875							23	27	31	61	.953125 .958333 .964286 .96875
.984375									3.	63	.984375

Decimal Equivalents of Fractions and Nearest Equivalent $^{\rm 64ths}$

Fr.	Decimal	Near- est 64th	Fr.	Decimal	Near- est 64th	Fr.	Decimal	Near- est 64th
$\begin{array}{c} \frac{1}{32} \\ \frac{1}{32} \\ \frac{1}{31} \\ \frac{1}{30} \\ \frac{1}{29} \\ \frac{1}{28} \\ \frac{1}{217} \\ \frac{1}{26} \end{array}$	0.0313 0.0323 0.0333 0.0345 0.0357 0.0370 0.0385	1/3/2	$\begin{array}{c} 2 \\ \overline{1} \\ \overline{5} \\ \underline{3} \\ \underline{2} \\ \underline{2} \\ \underline{7} \\ \underline{7} \\ \underline{4} \\ \underline{7} $	0.1333 0.1364 0.1379 0.1429 0.1481	- 1/8 - 9/64	$\begin{array}{c} 7 \\ 26 \\ 3 \\ 5 \\ \hline 1 \\ \hline 2 \\ 6 \\ \hline 3 \\ \hline 1 \\ \hline 2 \\ \hline 6 \\ \hline 3 \\ \hline 3 \\ \hline 1 \\ \hline 1 \\ \hline 2 \\ \hline 6 \\ \hline 3 \\ \hline 3 \\ \hline 1 \\ \hline 1 \\ \hline 1 \\ \hline 2 \\ \hline 6 \\ \hline 3 \\ \hline 3 \\ \hline 1 \\ \hline 1 \\ \hline 1 \\ \hline 2 \\ \hline 6 \\ \hline 3 \\ \hline 3 \\ \hline 1 \\ \hline 1 \\ \hline 1 \\ \hline 2 \\ \hline 6 \\ \hline 3 \\ \hline 3 \\ \hline 1 \\ \hline 1 \\ \hline 1 \\ \hline 2 \\ \hline 6 \\ \hline 3 \\ \hline 3 \\ \hline 1 \\ \hline 1 \\ \hline 1 \\ \hline 2 \\ \hline 6 \\ \hline 3 \\ \hline 3 \\ \hline 1 \\ \hline 1 \\ \hline 1 \\ \hline 2 \\ \hline 6 \\ \hline 3 \\ \hline 3 \\ \hline 1 \\ \hline 1 \\ \hline 1 \\ \hline 1 \\ \hline 2 \\ \hline 6 \\ \hline 6 \\ \hline 3 \\ \hline 1 \\ \hline 1 \\ \hline 1 \\ \hline 1 \\ \hline 2 \\ \hline 6 \\ \hline 6 \\ \hline 8 \\ \hline 1 \\ \hline 1 \\ \hline 1 \\ \hline 1 \\ \hline 1 \\ \hline 1 \\ \hline 2 \\ \hline 6 \\ \hline 6 \\ \hline 8 \\ \hline 1 \\ \hline $	0.2593 0.2609 0.2632 0.2667 0.2692 0.2727	17 64
26 125 14 124 123 121 120 19	0.0400 0.0417 0.0435 0.0455 0.0476	3 64	$ \begin{array}{c} 3 \\ 20 \\ 2 \\ 13 \\ 5 \\ 32 \\ \hline 3 \\ 2 \\ \hline 3 \\ 2 \\ 5 \\ 3 \\ 19 \\ 4 \\ 25 \\ 5 \\ 3 \\ 1 \end{array} $	0.1500 0.1538 0.1563 0.1579 0.1600 0.1613	5 3 2	$ \begin{array}{r} $	0.2759 0.2778 0.2800 0.2813 0.2857	9 3 2
$\frac{1}{18}$ $\frac{1}{17}$ $\frac{1}{16}$	0.0500 0.0526 0.0556 0.0589 0.0625	1 16	5 31 16 5 29 4 23 8 17 5 28	0.1667 0.1724 0.1739 0.1765 0.1785	11 64	$\begin{array}{c} \frac{9}{31} \\ \frac{7}{7} \\ \frac{7}{27} \\ \frac{5}{17} \\ \frac{8}{27} \\ \frac{3}{10} \\ \frac{7}{23} \end{array}$	0.2903 0.2917 0.2941 0.2963 0.3000 0.3043	19 64
2 31 15 2 29 1 14 2 27 13 25 12 25 11 12	0.0667 0.0690 0.0714 0.0740 0.0769	5 84	$\begin{array}{c} 2\\ \overline{111}\\ \underline{5}\\ \overline{277}\\ \underline{3}\\ \overline{16}\\ \underline{211}\\ \underline{5}\\ \underline{26}\\ \underline{6}\\ \overline{3}\\ \underline{1}\\ $	0.1852 0.1875 0.1905 0.1923 0.1935	3 16	$\begin{array}{c} 7\\ \hline 23\\ 4\\ \hline 18\\ 9\\ \hline 29\\ \hline \hline 5\\ \hline \hline 6\\ \hline 19\\ \hline 7\\ \hline 22\\ \hline \end{array}$	0.3077 0.3103 0.3125 0.3158 0.3182	5 16
$\frac{\frac{2}{23}}{\frac{1}{11}}$	0.0800 0.0833 0.0870 0.0909 0.0938 0.0952	3 2	156954 2572 41934 1587 272	0.2000 0.2069 0.2083 0.2105 0.2143 0.2174	1364	8 5 9 8 9 1 1 3 1 5 9 8 9 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1 1	0.3200 0.3214 0.3226 0.3333 0.3438	21 61 11 32
21 21 3 3 1 10 3 29 28 28 28 27 7	0.0968 0.1000 0.1034 0.1053 0.1071	7	$ \begin{array}{c c} \hline	0.2188 0.2222 0.2258 0.2273 0.2308	3 2	$\begin{array}{c} \frac{1}{2} \frac{0}{9} \\ \frac{9}{2} \frac{1}{6} \\ \frac{8}{2} \frac{3}{3} \\ 1 \frac{6}{11} \\ \frac{6}{11} \\ \frac{1}{3} \frac{1}{4} \end{array}$	0.3462 0.3478 0.3500 0.3529 0.3548	
$\frac{\frac{3}{2}}{5}$	0.1111 0.1154 0.1176 0.1200 0.1250	7 64 1 8	$\frac{17}{30}$ $\frac{4}{17}$ $\frac{1}{5}$ $\frac{1}{2}$ $\frac{6}{5}$ $\frac{7}{2}$ $\frac{9}{4}$	0.2333 0.2353 0.2381 0.2400 0.2414	154	$ \begin{array}{r} $	0.3571 0.3600 0.3636 0.3667 0.3684	23 64
$\frac{\frac{4}{31}}{\frac{3}{23}}$	0.1290 0.1304		$\frac{\frac{1}{4}}{\frac{8}{31}}$	0.2500	1 4	$\begin{array}{c} 7 \\ 1\overline{9} \\ \underline{10} \\ 2\overline{7} \\ \underline{38} \\ \end{array}$	0.3704 0.3750	38

 $\begin{array}{cccc} \textbf{Decimal Equivalents of Fractions and Nearest Equivalent} \\ & \textbf{64ths} \end{array}$

	i -		ī		T	1		
Fr.	Decimal	Near- est 64th	Fr.	Decimal	Near- est 64th	Fr.	Decimal	Near- est 64th
$\begin{array}{c} \frac{1}{2}\frac{1}{9} \\ \frac{8}{2}1 \end{array}$	0.3793 0.3810	38	$\frac{1}{3}\frac{6}{1}$ $\frac{1}{2}\frac{5}{9}$	0.5161	33 64	11 12 12 19	0.6296 0.6316	5/8
$\begin{array}{c} \frac{5}{13} \\ \frac{1}{3} \\ \frac{1}{3} \\ \frac{1}{7} \\ \frac{7}{18} \\ \frac{9}{23} \\ \frac{1}{28} \end{array}$	0.3846 0.3871 0.3889 0.3913	25 64	1615994735523 1223121212121212121212121212121212121	0.5185 0.5200 0.5217 0.5238		190 7 111 165 9 140 311 17	0.6333 0.6364 0.6400 0.6429	41 64
$\frac{23}{11}$ $\frac{2}{28}$	0.3929	64	11 10 19 9 17 17 17 32	0.5263		$\frac{14}{20}$ $\frac{31}{17}$	0.6452	
$\frac{\frac{2}{5}}{\frac{13}{32}}$ $\frac{11}{27}$	0.4063	$\frac{1}{3}\frac{3}{2}$	$ \begin{array}{r} \frac{17}{32} \\ \hline 8 \\ \hline 15 \\ \hline 28 \\ \end{array} $	0.5313 0.5333 0.5357	$\begin{array}{ c c }\hline \frac{1}{3}\frac{7}{2}\\ \hline \end{array}$	305/37/69/91/2 121/21/21/22/3	0.6500 0.6522 0.6538	
11 27 9 22 7 17 12 29	0.4091 0.4118 0.4138		$\frac{7}{13}$ $\frac{13}{24}$	0.5385			0.6552 0.6563	$\frac{2}{3}\frac{1}{2}$
$\frac{129}{5}$ $\frac{129}{133}$ $\frac{1}{3}$ $\frac{1}{1}$ $\frac{1}{2}$ $\frac{1}{6}$ $\frac{1}{3}$	0.4167 0.4194 0.4211	$\frac{27}{64}$	733346 117111069	0.5455 0.5484 0.5500	35 64	2 3 21 31 19 28	0.6667 0.6774 0.6786	43 64
126 37 130	0.4231 0.4286 0.4333		5 9 1 1 5 9 6	0.5517 0.5556 0.5600		1119 87 05 23 51 6	0.6800 0.6818 0.6842	
$\begin{array}{c} \frac{130}{300} \\ \frac{100}{23} \\ \hline 16 \\ \hline \frac{11}{25} \end{array}$	0.4348 0.4375	76	9 13 23 17 30	0.5625 0.5652 0.5667	9 16	16 20 29 13	0.6875 0.6897 0.6923	116
25 4 9 13 29	0.4400 0.4444 0.4483		4 7 15 26	0.5714 0.5769	2.5	$ \begin{array}{r} 13 \\ \frac{16}{23} \\ \hline 10 \end{array} $	0.6957 0.7000	
129 129 213 7 114	0.4500 0.4516 0.4545 0.4583	29 64	4½7-516-145-801-7-121-15-01-1-121-15-12-15-15-12	0.5789 0.5806 0.5833 0.5862	37 64	123 70972377421 12231	0.7037 0.7059 0.7083 0.7097	4 a 6 4
$\begin{array}{c} 24 \\ \hline 13 \\ 128 \\ \hline 28 \\ 7 \\ \hline 15 \\ \hline 3 \\ \hline 2 \end{array}$	0.4615 0.4642 0.4667		10 13 22 16	0.5882 0.5909 0.5926		$\frac{\frac{5}{7}}{\frac{2}{3}\frac{3}{2}}$	0.7143 0.7188	$\frac{23}{32}$
15 15 32 8 17 9 19	o.4688 o.4706	$\frac{1}{3}\frac{5}{2}$	3 5	o.5938 o.6000	19 32	185 138 181 219	0.7200 0.7222 0.7241	
9 1 9 1 0 2 1 1 1 2 3 2 3	0.4737 0.4762 0.4783		178 124 218 118 131 8 13	0.6071 0.6087 0.6111 0.6129	<u>39</u>	8 11961154977807	0.7273 0.7308 0.7333	4764
1211325374551 1212121212131	0.4800 0.4815 0.4828	$\frac{31}{64}$	$ \begin{array}{r} 31 \\ 8 \\ 13 \\ \hline 13 \\ \hline 121 \\ \hline 18 \\ \hline 29 \\ \hline $	0.6154 0.6190		17 17 23 20 7	0.7368 0.7391 0.7407	
$\frac{\frac{13}{31}}{\frac{1}{2}}$	0.4839 0.5000	$\frac{1}{2}$	1 8 2 9 5 8	0.6207 0.6250	<u>5</u> 8	$\frac{\frac{23}{31}}{\frac{3}{4}}$	0.7419 0.7500	34

Decimal Equivalents of Fractions and Nearest Equivalent $$64 \mathrm{ths}$$

Fr.	Decimal	Near- est 64th	Fr.	Decimal	Near- est 64th	Fr.	Decimal	Near- est 64th
239.945.6[1.3]7.8[0.0]87.[2.4]1 2[2.1]2.1[2.1]-2[3.1[1.4]2.2[8.7]9.2[8	0.7586 0.7600 0.7619 0.7647 0.7667	49 64	2 6 3 1 2 1 2 5 1 6 1 9 2 7 3 2 1 1 1 3	0.8387 0.8400 0.8421 0.8438 0.8462	27 32	1/123/524/35/r-3/474/9+4509/-15/6	0.9167 0.9200 0.9231 0.9259 0.9286	5 9 6 4
	0.7692 0.7727 0.7742 0.7778 0.7813 0.7826	2 5 3 2	137/035 122267 222122350371 22237 78	0.8500 0.8519 0.8571 0.8621 0.8636 0.8667	554		0.9310 0.9333 0.9355 0.9375 0.9412 0.9444	15 16
**\\\31\\45\\99\\48\\9\\75\\1\\67\\18\\6\\7\\18\\6\\7\\18\\6\\7\\18\\6\\7\\18\\6\\7\\18\\6\\7\\18\\6\\7\\18\\6\\7\\18\\6\\7\\18\\6\\7\\7\\7\\8\\8\\7\\7\\8\\8\\7\\7\\8\\8\	0.7857 0.7895 0.7917 0.7931 0.8000	5 <u>1</u> 64		0.8696 0.8710 0.8750 0.8800 0.8824	7/8	다. (- 1 - 1 - 1 - 1 - 1 - 1 - 1 - 1 - 1 -	0.9474 0.9500 0.9524 0.9545 0.9565	6 ±
231 226 127 136 227	0.8065 0.8077 0.8095 0.8125 0.8148 0.8182	13 16	지한 교실 (교육 이 교육 대한 대한 대한 대한 대한 대한 대한 대한 대한 대한 대한 대한 대한	o.8846 o.8889 o.8929 o.8947 o.8966 o.9000	57 64	44495566778884 422222222222222	0.9583 0.9600 0.9615 0.9630 0.9643 0.9655	
(3) - 130 4/- 9/94/9 (4) 9/H 2/24/H - (212/215/6	0.8182 0.8214 0.8235 0.8261 0.8276 0.8334	53 64	10 23 31 21 21 21 22 21 21 21 21 21 21 21 21 21	0.9000 0.9032 0.9048 0.9063 0.9091 0.9130	$\frac{2}{3}\frac{9}{2}$	2300 301 311 332 332 332	o.9667 o.9678 o.9688 1.0000	3 1 3 2

TABLE OF PRIME-NUMBER FRACTIONS

THE table shows decimal equivalents of common fractions having prime numbers for both numerator and denominator. As an example, suppose it is required to find the thread angle of a worm.

Tangent thread angle =
$$\frac{number\ threads}{diametral\ pitch\ \times\ pitch\ diameter}$$

Find the angle of a worm 7 diametral pitch, 5 threads, 2-inch P. diameter: Tangent angle = $\frac{5}{7} \times \frac{1}{2}$

Then from table

$$\frac{5}{7}$$
 = 0.7143 and $\frac{0.7143}{2}$ = 0.35715

which is the tangent for 10 degrees 21 minutes, nearly.

PRIME NUMBER FRACTIONS AND THEIR DECIMAL EQUIVALENTS Denominators (Prime Numbers Only)

	1		1	1						1		1	
		97	89	83	79	73	71	67	6 r	59	53	47	43
	-												
	1					.0137	.0141	.0149		.0169	.0189	.0213	.0233
	3	.0309	.0337	.0361	.0380	.0411	.0423	.0448		.0508	.0566	.0638	.0698
	5			.0602						.0847	.0943	.1064	
_	7			.0843					.1148	.1186	.1321	.1489	.1628
2				.1325			.1549	.1642	.1803	.1864	.2075	.2340	
ű	13	.1340	.1461	.1566	.1646		.1831	.1940	.2131	.2203	.2453	.2766	
Š				.2048		.2329			.2787	.2881	.3208	.3617	
Numbers only)				.2289			.2676	.2836	.3115	.3220	.3585	.4043	
Ē				.2771		.3151	.3299	·3433	.3770	.3898	.4340	.4894	
Ξ				.3494		.3973	.4085	.4328	-4754	.4915	.5472	6170	
				-3735		.4247	.4366	.4627	.5082	·5254	.5849	.6596	
ne	37	.3814	.4157	.4458	.4084	.5068	.5211	.5522	.6066	.6271	.6981	.7872	
(Prime	41	.4227	.4007	.4940	.5190		-5775	.6119	.6721	.6949	.7736	.8723	.9535
<u>-</u>			.4031	.5181	.5443	.5890	.6056 .6620	.6418	.7049	.7288	.8113	.9149	
Ś				.5663		.6438 .7260	.7465	.7015	.7705	.7966	.8868		
Numerators	53	.5404	.5955	.6386 .7108	.0700	.8082	.8310	.7910 .8806	.8689	.8983			
ra	29	6080	6874	7349	.7400	.8356	.8592		.9672				
ne	67	6007	7528	.8072	8487	.9178	.9437	.9104					
_≅	27	7220	7078	.8554	8087	.9726	.9437						
Z				.8795		.9/20							
	70	8144	.8876	0518	.9241								
		.8557		.9310	. 1	1						i	
		.9175	.9320		1	1	i						
	109	9-13					1						

Denominators (Prime Numbers Only)

		41	37	31	29	23	19	17	13	11	7	5	3
ttors (Prime bers only)	3 5 7	.0732 .1220 .1707	.1892	.0968 .1613 .2258	.1034 .1724 .2414	.1304 .2174 .3043	.1579 .2632 .3684	.4118 .6471 .7647		.2727 .4545	.4286 .7143		
Numer Num	23 29 31	.4634 .5610 .7073 .7561 .9024	.5135 .6212 .7838 .8778	.6129 .7419 .9355	.6552 .7931	.5652 .7391 .8261	Only number inator by sin	ers for	both tren in t	he nun able.	nerator Others	naving and de can be ion.	enom-

Equivalents of Inches and Fractions of Inches in Decimals of a Foot

o In.	r In.	2 In.	3 In.	4 In.	5 In.
.0026 .0052 .0078	.0833 .0859 .0885	.1667 .1693 .1719 .1745	.2500 .2526 .2552 .2578	·3333 ·3359 ·33 ⁸ 5 ·3411	.4167 .4193 .4219 .4245
.0104	.0938	.1771	.2604	•3438	.4271
.0130	.0964	.1797	.2630	•3464	.4297
.0156	.0990	.1823	.2656	•3490	.4323
.0182	.1016	.1849	.2682	•3516	.4349
.0208	.1042	.1875	.2708	•3542	•4375
.0234	.1068	.1901	.2734	•3568	•4401
.0260	.1094	.1927	.2760	•3594	•4427
.0286	.1120	.1953	.2786	•3620	•4453
.0313	.1146	.1979	.2813	.3646	•4479
.0339	.1172	.2005	.2839	.3672	•4505
.0365	.1198	.2031	.2865	.3698	•4531
.0391	.1224	.2057	.2891	.3724	•4557
.0417	.1253	.2083	.2917	•3750	.458 3
.0443	.1276	.2091	.2943	•3776	.4609
.0469	.1302	.2135	.2969	•3802	.4635
.0495	.1328	.2161	.2995	•3828	.4661
.0521	.1354	.2188	•3021	•3854	.4688
.0547	.1380	.2214	•3047	•3880	.4714
.0573	.1406	.2240	•3073	•3906	.4740
.0599	.1432	.2266	•3099	•3932	.4766
.0625	.1458	.2292	.3125	.3958	.4792
.0651	.1484	.2318	.3151	.3984	.4818
.0677	.1510	.2344	.3177	.4010	.4844
.0703	.1536	.2370	.3203	.4036	.4870
.0729 .0755 .0781	.1563 .1589 .1615	.2396 .2422 .2448 .2474	.3229 .3255 .3281 .3307	.4063 .4089 .4115 .4141	.4896 .4922 .4948 .4974
	.0026 .0052 .0078 .0104 .0130 .0156 .0182 .0208 .0234 .0260 .0286 .0313 .0339 .0365 .0391 .0417 .0443 .0469 .0495 .0521 .0547 .0573 .0599 .0625 .0651 .0677 .0703	.0833 .0026 .0859 .0052 .0885 .0078 .0911 .0104 .0938 .0130 .0964 .0156 .0990 .0182 .1016 .0208 .1042 .0234 .1068 .0260 .1094 .0286 .1120 .0313 .1146 .0339 .1172 .0365 .1198 .0391 .1224 .0417 .1253 .0443 .1276 .0469 .1302 .0495 .1328 .0521 .1354 .0547 .1380 .0573 .1406 .0599 .1432 .0625 .1458 .0651 .1484 .0677 .1510 .0703 .1536	.0833 .1667 .1693 .0052 .0885 .1719 .0078 .0911 .1745 .1745 .0104 .0938 .1771 .0130 .0964 .1797 .0156 .0990 .1823 .0182 .1016 .1849 .0208 .1042 .1083 .1091 .0260 .1094 .1927 .0286 .1120 .1053 .0313 .1146 .1979 .0339 .1172 .2005 .0365 .1198 .2031 .0391 .1224 .2057 .0417 .1253 .2083 .0493 .1324 .2057 .0417 .1253 .2083 .0443 .1276 .2091 .0469 .1302 .2135 .0495 .1328 .2101 .0521 .1354 .2188 .0547 .1380 .2214 .0573 .1406 .2240 .0599 .1432 .2266 .2344 .0573 .1496 .2240 .0599 .1432 .2266 .0625 .1458 .2292 .0651 .1484 .2318 .0677 .1510 .2344 .0703 .1536 .2370 .0729 .1563 .2396 .0755 .1589 .2422 .0781 .1615 .2448	.0833 .1667 .2500 .0026 .0859 .1693 .2526 .0052 .0885 .1719 .2552 .0078 .0911 .1745 .2578 .0104 .0938 .1771 .2604 .0130 .0964 .1797 .2630 .0156 .0990 .1823 .2656 .0182 .1016 .1849 .2682 .0208 .1042 .1875 .2708 .0208 .1042 .1875 .2708 .0234 .1068 .1901 .2734 .0260 .1094 .1927 .2760 .0286 .1120 .1953 .2786 .0313 .1146 .1979 .2813 .0339 .1172 .2005 .2839 .03365 .1198 .2031 .2865 .0391 .1224 .2057 .2891 .0417 .1253 .2083 .2917 .0443 .1276 .2091 .2943 .0469 .1302 .2135 .2969 .0495 .1328 .2161 .2995 .0521 .1354 .2188 .3021 .0547 .1380 .2214 .3047 .0573 .1406 .2240 .3073 .0599 .1432 .2266 .3099 .0625 .1458 .2292 .3125 .0651 .1484 .2318 .3151 .0677 .1510 .2344 .3177 .0703 .1536 .2370 .3203	.0833 .1667 .2500 .3333 .0026 .0859 .1693 .2526 .33359 .0052 .0885 .1719 .2552 .3385 .0078 .0911 .1745 .2578 .3411 .0104 .0938 .1771 .2604 .3438 .0130 .0964 .1797 .2630 .3464 .0156 .0990 .1823 .2656 .3490 .0182 .1016 .1849 .2682 .3516 .0208 .1042 .1875 .2708 .3542 .0234 .1068 .1901 .2734 .3568 .0260 .1094 .1927 .2760 .3594 .0286 .1120 .1953 .2786 .3620 .0313 .1146 .1979 .2813 .3646 .0286 .1120 .1953 .2786 .3620 .0313 .1146 .1979 .2813 .3646 .0339 .1172 .2005 .2839 .3672 .0365 .1198 .2031 .2865 .3698 .0391 .1224 .2057 .2891 .3724 .0417 .1253 .2083 .2917 .3750 .0469 .1302 .2135 .2969 .3802 .0495 .1328 .2161 .2995 .3828 .0573 .1406 .2240 .3073 .3984 .0573 .1406 .2240 .3073 .3906 .0599 .1432 .2266 .3099 .3932 .0625 .1458 .2292 .3125 .3998 .05573 .1406 .2240 .3073 .3906 .0599 .1432 .2266 .3099 .3932 .0625 .1458 .2292 .3125 .3958 .0651 .1484 .2318 .3151 .3984 .0677 .1510 .2344 .3177 .4010 .0703 .1536 .2370 .3203 .4036 .0729 .1563 .2396 .3229 .4063 .0755 .1589 .2422 .3255 .4089 .0755 .1589 .2422 .3255 .4089 .0755 .1589 .2422 .3255 .4089 .0755 .1589 .2422 .3255 .4089 .0755 .1589 .2422 .3255 .4089 .0751 .1615 .2448 .3281 .4115

EQUIVALENTS OF INCHES AND FRACTIONS OF INCHES IN DECIMALS

τ _n .	6 In.	7 In.	8 In.	9 In.	10 In.	11 In.
3 2 1 1 6 3 2 3 2	-5000 -5026 -5052 -5078	•5833 •5859 •5885 •5911	.6667 .6693 .6719 .6745	.75°° .7526 .7552 .7578	.8333 .8359 .8385 .8411	.9167 .9193 .9219 .9245
1 8 5 3 2 1 6 7 3 2 3 2 3 2 3 2	.5104 .5130 .5156 .5182	.5938 .5964 .5990 .6016	.6771 .6797 .6823 .6849	.7604 .7630 .7656 .7682	.8438 .8464 .8490 .8516	.9271 .9297 .9323 .9349
$ \begin{array}{c} $.5208 .5234 .5260 .5286	.6042 .6068 .6094 .6120	.6875 .6901 .6927 .6953	.7708 .7734 .7760 .7786	.8542 .8568 .8594 .8620	•9375 •9401 •9427 •9453
$\frac{3}{8}$ $\frac{1}{3}$ $\frac{3}{2}$ $\frac{7}{16}$ $\frac{1}{3}$ $\frac{5}{2}$	•5313 •5339 •5365 •5391	.6146 .6172 .6198 .6224	.6979 .7005 .7031 .7057	.7813 .7839 .7865 .7891	.8646 .8672 .8698 .8724	•9479 •9505 •9531 •9557
$\begin{array}{c} 1 \\ 2 \\ 1 \\ 7 \\ 2 \\ 9 \\ 1 \\ 9 \\ 6 \\ 1 \\ 9 \\ 3 \\ 2 \\ \end{array}$	•5417 •5443 •5469 •5495	.6250 .6276 .6302 .6328	.7083 .7109 .7135 .7161	.7917 .7943 .7969 .7995	.8750 .8776 .8802 .8828	.9583 .9609 .9635 .9661
58 281 31 10 31 21 31 21 31 21 31 31	•5521 •5547 •5573 •5599	.6354 .6380 .6406 .6432	.7188 .7214 .7240 .7266	.8021 .8047 .8073 .8099	.8854 .8880 .8906 .8932	.9688 .9714 .9740 .9766
3 4 5 243 67 21 3 4 5 243 67 21	•5625 •5651 •5677 •5703	.6458 .6484 .6510 .6536	.7292 .7318 .7344 .7370	.8125 .8151 .8177 ,8203	.8958 .8984 .9010 .9036	.9792 .9818 .9844 .9870
7 8 9 215 61 21 2 31 13 3	•5729 •5755 •5781 •5807	.6563 .6589 .6615 .6641	.7396 .7422 .7448 .7474	.8229 .8255 .8281 .8307	.9063 .9089 .9115 .9141	.9896 .9922 .9948 .9974
=-			1	1		

		Added Fractions to Sixty-fourths
		ਜੇਤ ਗੁਰੂ ਪੁਰੂ ਮੁਤੂ ਨੂੰ ਨੂੰ ਜੋੜ ਜ਼ਿਲ੍ਹ ਉਰ ਮੁਤੂ ਹਰ ਜਾਂ ਨਰ ਉਰ ਮੁਤੂ ਜੂੜ ਗੁਰੂ ਪੁਰੂ ਮੁਤੂ ਨੂੰ ਜੋੜ ਜ਼ੁਰੂ ਜ਼ੁਰੂ ਜ਼ੁਰੂ ਜ਼ੁਰੂ ਜ਼ੁਰੂ ਗੁਰੂ ਹਨ ਹਨ ਹਨ ਹਨ ਹਨ
TABLE	0	0000000 0000244 000077 0003906 0003906 0003906 001958 001975 0019
I. — SQUARE	П	1.00000 1.03149 1.03148 1.00348 1.10235 1.10235 1.10235 1.10235 1.26563 1.3091 1.3091 1.31399 1.3139 1.3139 1.3139 1.3139 1.31399 1.31399 1.31399 1.31399
S OF NUMBE	2	4.000000 4.00274 4.12503 4.12504 4.13800 4.38160 4.38160 4.38160 4.38160 4.38170 4.11704 4.71704 4.71704 4.71704 4.71704 6.2023 6.20230 6.20230 6.203
Table I.—Squares of Numbers from "o"	ю	9,00000 9,00399 9,18848 9,18845 9,17850 9,47485 9,56821 9,56821 9,56821 0,76563 9,56323 0,26001 10,26001 10,26001 10,26001 10,26001 10,26001 10,26001 10,26001 11,2600
to " 764"	4	16,00000 16,1224 16,2304 16,3702 16,5301 16,5311 17,1478 17,14478 17,14478 17,14478 17,14478 17,14478 17,14478 17,14478 17,1478 18,147
BY 64THS — NUMBER	w	25.00000 25.115448 25.115495 25.015095 25.02501 25.02501 25.02501 26.02501 26.02501 26.02501 26.02501 26.02501 27.03535 27.03501 27.02501
NUMBER	9	36.00000 36.18774 36.37593 36.56470 37.1336 37.32446 37.72728 37.70728 37.7
	7	40.00000 40.21894 40.4845 40.6845 40.87896 50.0995 50.0995 50.0985 50.0985 50.0985 50.0985 50.0885 51.46000 51.66016 51.88501 52.11035 52.11035 52.11035 52.11035 52.11035 52.11035 52.11035 53.4438 54.5040 54.5040 54.5040 54.5040 55.6421 55.6421 56.6421

1. 250322 2.11157 6.01782 11.03407 10.3332 2.11157 5.01773 11.03407 10.03407 12.03233 4.14473 5.554007 5.554007 5.254019 5.254019 2.24312 2.14772 2.14773 12.03233 12.03007 3.02500 2.25000 3.02500 <th></th> <th></th> <th>_</th> <th></th> <th></th> <th>_</th> <th></th> <th></th> <th>5</th> <th>цр</th> <th>no</th> <th>Į-</th> <th>£λ</th> <th>xi</th> <th>3 0</th> <th>1</th> <th>suo</th> <th>itoı</th> <th>31.</th> <th>I</th> <th>əį</th> <th>p</th> <th>٧</th> <th></th> <th>-</th> <th></th> <th></th> <th></th> <th></th> <th></th> <th></th> <th></th> <th></th>			_			_			5	цр	no	Į-	£λ	xi	3 0	1	suo	itoı	31.	I	əį	p	٧		-								
205322 2.11157 6.01782 11.02407 19.89932 29.73657 41.04282 2.24619 2.20337 6.04773 12.04967 2.90973 44.04712 2.25580 2.229712 6.04773 12.23902 20.23008 30.2500 2.25580 2.29712 6.42723 12.23902 20.23008 47.2500 2.25580 2.29712 6.42723 12.23902 20.23008 47.25003 2.20580 2.2473 12.24073 12.24073 47.4413 47.4413 2.2000 2.2473 12.24073 20.24500 47.2500 47.2500 3.34229 2.24040 6.25023 12.24073 20.44141 43.04774 3.71236 2.40046 6.40723 12.24073 21.145073 47.45374 3.71236 2.40046 6.70720 12.24073 21.14507 47.45330 3.71236 2.54004 6.72754 12.24073 31.46504 47.4772 3.72530 2.54004 6.80063 13.2416 47.4772	- 1000 1000	-40		-103 -103		o/e	,	O:10		wałso		-121 -121 -121		19		csta table		6 4		6460 PD 03		n ::		100		z- a0		040 104	;	es es	5	103	
2.1057 6.01782 11.02407 19.83932 29.73657 41.64282 2.2033 6.09473 12.03223 20.009073 30.07937 41.64473 2.2034 6.24807 12.25900 20.25000 30.25000 42.25000 2.29712 6.24807 12.35902 20.39087 30.4212 42.4537 2.2972 6.24807 12.25000 20.39087 30.4212 42.4537 2.2972 6.24807 12.25000 20.25000 30.25002 42.25002 2.2972 6.24073 12.26073 20.01941 20.01941 20.01941 20.01941 42.65773 42.4537 2.2904 6.25011 12.01024 20.01941 20.01941 30.01141 43.01573 42.4537 42.4537 2.2904 6.25011 12.01024 20.01902 30.25323 30.501141 30.01141 43.01841 43.21773 42.45377 42.45377 42.45017 42.45017 42.45017 42.45017 42.45017 42.45017 42.45017 42.45017 42.4	64 44		6.0 6.4		80 C		ea Fi4	;	w.a ⊕.4		##3 ##3	-	40		4.0		4.4		4 D		4.4		9 to		00 04		4.5	;	7.4		44	6.0	4
6.001782 11.02407 10.83032 29.73657 41.64282 6.00473 12.03233 10.09073 3.04073 41.64173 6.000473 12.03233 10.09073 3.040973 41.64173 6.25000 12.25000 20.25000 30.25000 42.25000 12.35002 20.39087 30.24212 42.4537 42.64073 20.051023 30.05473 42.65723 6.48073 12.48073 20.051023 30.05473 42.86177 12.60141 20.051023 30.05473 42.86177 12.01544 13.025473 42.86177 12.01544 13.025473 42.86177 12.01544 13.025473 42.86177 12.01544 13.025473 42.86177 12.01544 13.025473 42.86177 12.01544 13.025473 42.86177 12.01544 13.025473 42.86177 12.01544 13.025473 42.86177 12.01544 13.025473 42.86177 12.01544 13.025473 42.86177 12.01544 13.025473 42.86177 12.01544 13.025473 42.86177 12.01544 13.025473 42.86177 12.01544 13.025473 42.86177 12.015477 42.015473 42.86177 12	.205322 .219726 .234619	.250000	.265869	.282227	.299072	.316406	.334229	.352539	.371338	.390625	.410400	.430654	.451416	.472656	.494385	.\$16602	.539307	.502500	.586182	.616352	.535010	.060150	162580.	.7 II 914	.738526	.765625	.793213	.821289	.849854	906878.	.908447	-938477	.900994
11.02407 10.83032 20.73657 41.64282 12.03223 10.090973 20.090973 41.84473 12.03233 10.090973 20.090972 20.25000 12.55902 20.25000 30.25000 42.25000 12.55902 20.25000 30.25000 42.25001 42.00141 42.00	2.11157 2.15723 2.20337	2.25000	2.29712	2.34473	2.39282	2.44141	2.49048	2.54004	2.59009	2.64063	2.69165	2.74316	2.79517	2.84766	2.90063	2.95410	3.00806	3.06250	3.11743	3.17285	3.22876	3.28516	3.34204	3.3994I	3.45728	3.51563	3.57446	3.63779	3.69360	3.75391	3.81470	3.87598	3.93774
19.83932 29.73657 41.64282 20.090973 20.73657 41.64173 20.109092 30.29733 42.04712 20.25000 30.25000 42.25000 20.39087 30.24212 42.45537 20.501641 30.50473 42.86157 20.50261 30.50473 42.86157 20.50312 30.50473 42.86157 20.50543 30.50473 42.86157 20.50543 31.20904 43.60511 21.1023 31.20904 43.06517 21.1024 31.40509 43.06534 21.6800 31.90316 44.3063 21.6800 31.90316 44.3050 21.6800 31.90316 44.3050 21.6800 32.3470 44.3360 22.2600 32.3470 44.3360 22.2600 32.3470 45.3160 22.2600 32.3430 45.3160 22.2600 32.3030 45.3563 22.2600 33.0037 45.323 22.31001 </th <td>6.01782 6.09473 6.17212</td> <td>6.25000</td> <td>6.32837</td> <td>6.40723</td> <td>6.48657</td> <td>0.56641</td> <td>6.64673</td> <td>6.72754</td> <td>6.80884</td> <td>6.89063</td> <td>6.97290</td> <td>7.05500</td> <td>7.13892</td> <td>7.22206</td> <td>7.30688</td> <td>7.39160</td> <td>7.47681</td> <td>7.56250</td> <td>7.64868</td> <td>7.73535</td> <td>7.82251</td> <td>7.91016</td> <td>7.99829</td> <td>8.08691</td> <td>8.17603</td> <td>8.26563</td> <td>8.35571</td> <td>8.44629</td> <td>8.53735</td> <td>8.62891</td> <td>8.72095</td> <td>8.81348</td> <td>8.90049</td>	6.01782 6.09473 6.17212	6.25000	6.32837	6.40723	6.48657	0.56641	6.64673	6.72754	6.80884	6.89063	6.97290	7.05500	7.13892	7.22206	7.30688	7.39160	7.47681	7.56250	7.64868	7.73535	7.82251	7.91016	7.99829	8.08691	8.17603	8.26563	8.35571	8.44629	8.53735	8.62891	8.72095	8.81348	8.90049
29,73657 41,64282 39,07367 39,07357 41,84473 39,07837 42,23000 30,42122 44,45,173 30,07672 30,4411 31,115,43 31,0004 31,100316	11.92407 12.03223 12.14087	12.25000	12.35962	12.46973	12.58032	12.69141	12.80298	12.91504	13.02759	13.14063	13.25415	13.30810	13.48267	13.59766	13.71313	13.82910	13.94556	14.06250	14.17993	14.29785	14.41626	14.53516	14.05454	14.77441	14.89478	15.01563	15.13696	15.25879	15.38110	15.50391	15.02720	15.75098	15.87524
41.64282 41.84473 42.25000 42.45357 42.65723 42.865723 42.865723 42.865723 42.865723 43.47773 43.47773 44.007,000 44.007,000 44.007,000 44.007,000 44.007,000 45.55625 45.41116 45.55625 45.4116 45.5625 46.41016 46.41016 46.41016 46.41016 46.41016 46.41016 47.20563 47.2	19.83032 19.96973 20.10962	20.25000	20.39087	20.53223	20.67407	20.81641	20.05923	21.10254	21.24634	21.39063	21.53540	21.68000	21.82042	21.97266	22.11938	22.26600	22.4143I	22.56250	22.71118	22.86035	23.01001	23.16016	23.31079	23.40191	23.61353	23.76563	23.91821	24.07129	24.22485	24.37891	24.53345	24.02242	24.84399
	29.73657 29.90723 30.07837	30.25000	30.42212	30.59473	30.76782	30.04141	31.11548	31.29004	31.46509	31.64063	31.81065	31.99316	32.17017	32.34760	32.52563	32.70410	32.58306	33.06250	33.24243	33.42285	33.00370	33.78516	33.96704	34.14941	34.33228	34.51563	34.69940	34.88379	35.00800	35.25391	35.43970	35.02598	35.01274
\$55.54907 \$55.7822 \$50.7822 \$50.7822 \$50.7923 \$5	41.64282 41.84473 42.04712	42.25000	42.45337	42.65723	42.86157	43.066.41	43.27173	43-47754	43.68384	43.89063	44.09790	. 44.30560	44.51392	44.72200	44.93188	45.14160	45.35181	45.56250	45.77368	45.98535	46.19751	46.41016	46.62329	48.83091	47.05103	47.26563	47.48071	47.69629	47.91235	48.12891	48.34595	48.503.40	40.70149
	55.549 55.782 56.015	56.2500	56.4840	56.7197	56.0553	57.101.4	57.4270	57.6650	57.9025	58.1406	58.3791	58.6181	58.8576	59.0976	59.3381	59.5791	59.8205	60.0625	60.3049	60.5478	50.7912	61.0351	01.2795	01.5244	61.7697	62.0156	62.2019	62.5087	02.7501	63.0035	03.2522	03.5000	03.7502

Decimal Equivalents, Squares, Square Roots, Cubes and Cube Roots of Fractions; Circumferences and Areas of Circles from $\frac{1}{64}$ to 1 inch

Frac- tion	Dec. Equiv.	Square	Sq. Root	Cube	Cube Root	Circum. Circle	Area Circle
$\begin{array}{c} 1 \\ 6 \overline{4} \\ 3 \overline{2} \\ \overline{6} \overline{4} \\ 1 \overline{6} \end{array}$.015625 .03125 .046875 .0625	.000244 .0009765 .002197 .003906	.1250 .1768 .2165 .2500	.000003815 .00003052 .000103	.2500 .3150 .3606 .3968	.04909 .09818 .1473 .1963	.000192 .000767 .001726
$\frac{\frac{5}{64}}{\frac{3}{32}}$ $\frac{7}{64}$ $\frac{1}{8}$.078125 .09375 .109375 .1250	.006104 .008789 .01196 .01563	.2795 .3062 .3307 .3535	.0004768 .0008240 .001308 .001953	.4275 .4543 .4782 .5000	.2455 .2945 .3436 .3927	.004794 .006903 .009396
$\begin{array}{c} 9\\ \overline{6} \\ \overline{4}\\ 5\\ \overline{3} \\ \overline{2} \\ 1\\ \overline{4} \\ \overline{4} \\ \overline{6} \\ \overline{4} \\ \overline{6} \\ \overline{6} \\ \end{array}$.140625 .15625 .171875 .1875	.01978 .02441 .02954 .03516	.3750 .3953 .4161 .4330	.002781 .003815 .005078 .006592	.5200 .5386 .5560 .5724	.4438 .4909 .5400 .5890	.01553 .01916 .02321 .02761
$\frac{13}{64}$ $\frac{7}{315}$ $\frac{1}{64}$.203125 .21875 .234375 .2500	.04126 .04786 .05493 .0625	.4507 .4677 .4841 .5000	.008381 .01047 .01287 .01562	.5878 .6025 .6166 .6300	.6381 .6872 .7363 .7854	.03241 .03758 .04314 .04909
$\begin{array}{c} \frac{17}{64} \\ \frac{9}{32} \\ \frac{19}{64} \\ \overline{16} \end{array}$.265625 .28125 .296875 .3125	.07056 .07910 .08813 .09766	.5154 .5303 .5449 .5590	.01874 .02225 .02616 .03052	.6428 .6552 .6671 .6786	.8345 .8836 .9327 .9817	.05541 .06213 .06922 .07670
214 112 234 334 8	.328125 ·34375 ·359375 ·375°	.1077 .1182 .12913 .1406	.5728 .5863 .5995 .6124	.03533 .04062 .04641 .05273	.6897 .7005 .7110 .7211	1.031 1.080 1.129 1.178	.08456 .09281 .1014 .1104
$\frac{25 4}{632}$ $\frac{327}{4}$ $\frac{4}{7}$ $\frac{7}{6}$.390625 .40625 .421875 .4375	.1526 .1650 .17800 .1914	.6250 .6374 .6495 .6614	.05960 .06705 .07508 .08374	.7310 .7406 .7500 .7592	1.227 1.276 1.325 1.374	.1226 .1296 .1398 .1503
9 4±5 221 4 1 333 6	.453125 .46875 .484375 .5000	.2053 .2197 .2346 .2500	.6732 .6847 .6960 .7071	.00304 .1030 .1136 .1250	.7681 .7768 .7853 .7937	1.424 1.473 1.522 1.571	.1613 .1726 .1843 .1963

Decimal Equivalents, Squares, Square Roots, Cubes, Cube Roots of Fractions; Circumferences and Areas of Circles from $^{1}_{64}$ to 1 inch.

Frac- tion	Dec. Equiv.	Square	Sq. Root	Cube	Cube Root	Circum. Circle	Area Circle
$\frac{3}{6}\frac{3}{4}$ $\frac{1}{1}\frac{7}{2}$ $\frac{3}{6}\frac{5}{4}$ $\frac{9}{1}$.515625 .53125 .546875 .5625	.2659 .2822 .2991 .3164	.7181 .7289 .7395 .7500	.1371 .1499 .1636 .1780	.8019 .8099 .8178 .8255	1.620 1.669 1.718 1.767	.2088 .2217 .2349 .2485
374 1923 365 8	.578125 .59375 .609375 .6250	·334 ² ·35 ² 5 ·37 ¹ 3 ·3906	.7603 .7706 .7806 .7906	.1932 .2093 .2263 .2441	.8331 .8405 .8478 .8550	1.816 1.865 1.914 1.963	.2625 .2769 .2916 .3068
41 64 241 323 464 116	.640625 .65625 .671875 .6875	.4104 .4307 .4514 .4727	.8004 .8101 .8197 .8292	.2629 .2826 .3°33 .325°	.8621 .8690 .8758 .8826	2.013 2.062 2.111 2.160	.3223 .3382 .3545 .3712
462934634	.703125 71875 734375 7500	.4944 .5166 .5393 .5625	.8385 .8478 .8569 .8660	.3476 .3713 .3961 .4219	.8892 .8958 .9022 .9086	2.209 2.258 2.307 2.356	.3883 .4057 .4236 .4418
4652214436 235611	.765625 .78125 .796875 .8125	.5862 .6104 .6350 .6602	.8750 .8839 .8927 .9014	.4488 .4768 .5060 .5364	.9148 .9210 .9271 .9331	2.405 2.454 2.503 2.553	.4604 .4794 .4987 .5185
564772554 23554 78	.828125 .84375 .859375 .8750	.6858 .7119 .7385 .7656	.9100 .9186 .9270 .9354	.5679 .6007 .6347 .6699	.9391 .9449 .9507 .9565	2.602 2.651 2.700 2.749	.5386 .5592 .5801 .6013
56492394456 2356456	.890625 .90625 .921875 .9375	.7932 .8213 .8499 .8789	.9437 .9520 .9601 .9682	.7064 .7443 .7835 .8240	.9621 .9677 .9732 .9787	2.798 2.847 2.896 2.945	.6230 .6450 .6675 .6903
614 312 634 1	.953125 .96875 .984375	.9084 .9385 .9690	.9763 .9843 .9922 I	.8659 .9091 .9539	.9841 .9895 .9948	2.994 3.043 3.093 3.1416	.7135 .7371 .7610 .7854

SQUARES, CUBES, SQUARE AND CUBE ROOTS OF NUMBERS FROM 1 TO 1000

1 2 3 3 4 4 5 5 6 7 8 9 10 11 12 13 11 14 15 16 11 7 18 19 20 20 21 22 23 22 4 25 26 27	1 4 9 16 25 36 49 64 81	1 8 27 64 125, 216 343 512	1.0000 1.4142 1.7321 2.0000 2.2361 2.4495	1.0000 1.2599 1.4422 1.5874	51 52 53	2601 2704	132651	7.1414	3.7084
3 4 5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26	9 16 25 36 49 64 81	27 64 125 216 343 512	1.7321 2.0000 2.2361	I.4422		2701	× . = 6 = 0		
4 5 6 7 8 9 10 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26	16 25 36 49 64 81	64 125, 216 343 512	2.0000 2.2361		E 2		140608	7.2111	3.7325
5 6 7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26	25 36 49 64 81	125. 216 343 512	2.2361	1.5874		2800	148877	7.2801	3.7563
7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26	36 49 64 81	216 343 512			54	2916	157464	7.3485	3.7798
7 8 9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26	49 64 81	343 512	~.4490	1.7100	55 56	3025 3136	166375 175616	7.4162	3.8030
9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26	64 81	512	2.6458	1.0171	57	3249	185193	7.5408	3.8485
9 10 11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26	81		2.8284	2.0000	58	3364	105112	7.6158	3.8700
11 12 13 14 15 16 17 18 19 20 21 22 23 24 25 26	100	729	3.0000	2.0801	59	3481	205379	7.6811	3.8930
12 13 14 15 16 17 18 19 20 21 22 23 24 25 26		1000	3.1623	2.1544	00	3600	216000	7.7460	3.9149
13 14 15 16 17 18 19 20 21 22 23 24 25 26	121	1331	3.3166	2.2240	6 r	3721	226981	7.8102	3.9365
1.4 1.5 1.6 1.7 1.8 1.9 2.0 2.1 2.2 2.3 2.4 2.5 2.6	1.44	1728	3.4641	2.2894	62	3844	238328	7.8740	3.9579
15 16 17 18 19 20 21 22 23 24 25 26	169	2197	3.6050	2.3513	63	3969	250047	7.9373	3.9791
16 17 18 19 20 21 22 23 24 25 26	196	2744	3.7417	2.4101	64	4096	262144	8.0000	4.0000
17 18 19 20 21 22 23 24 25 26	225 256	3375 4006	3.8730	2.4662	66	4225 4356	27 46 25 287 496	8.1240	4.0207
18 19 20 21 22 23 24 25 26	289	4913	4.1231	2.5713		4489	300763	8.1854	4.0615
21 22 23 24 25 26	324	5832	4.2426	2.6207	67	4624	314432	8.2462	1.0817
2 I 2 2 2 3 2 4 2 5 2 6	361	6859	4.3589	2.6684	69	4761	328500	8.3066	4.1016
22 23 24 25 26	400	8000	4.4721	2.7144	70	4900	3.13000	8.3666	4.1213
22 23 24 25 26	441	0261	4.5826	2.7589	71	5041	357911	8.4261	4.1408
24 25 26	484	10648	4.6004	2.8020	7.2	5184	373248	8, 1853	4.1602
25 26	529	12167	4.7958	2.8430	7.3	5329	389017	8.5440	4.1793
26	576	13824	4,8900	2.8845	7.4	5476	405224	8.6023	4.1983
	625	15025	5.0000	2.0240	7.5	5625	421875	8.6603	4.2172
2/	676	17570	5.0000	2.0625	76	5776	438976	8.7178	4.2358
28	729 784	19583 21952	5.1962	3.0000 3.0366	77 78	5929 6084	45 ⁶ 53.3 474552	8.8318	4.2727
20	841	24389	5.3852	3.0723	79	6241	493039	8.8882	1.2008
30	900	27000	5.4772	3.1072	80	6400	51 2000	8.9443	4.3089
31	961	20701	5.5678	3.1.11.1	81	6561	531441	9.0000	4.3267
32	1024	32768	5.6569	3.1748	82	6724	551368	9.0554	4.3445
33	1089	35937	5.7446	3.2075	83	6889	571787	9.1104	4.3621
34	1156	39304	5.8310	3.2396	84	7056	592704	9.1652	4.3795
35	1225	42875	5.9161	3.2711	85 86	7225	614125	9.2195	4.3968
36	1296 1369	46656 50653	6.0000	3.3019		7396 7569	636056 658503	9.2736	4.4140
37 38	1444	54872	6.16.14	3.3620	87 88	7744	681472	0.3808	4.4480
39	1521	59319	6.2450	3.3012	80	7021	704969	9.4340	4.4647
40	1600	64000	6.3246	3.4200	90	8100	729000	9.4868	4.4814
41	1681	68921	6.4031	3.4482	oı	8281	753571	9.5394	4.4979
12	1764	74088	6.4807	3.4760	92	8464	778688	9.5917	4.5144
43	1849	79507	6.5574	3.5034	93	8649	804357	9.6437	4.5307
44	1936	85184	6.6332	3.5303	94	8836	830584	9.6954	4.5468
45	2025	91125	6.7082	3.5569	95	9025	857375	9.7468	4.5629
46	2116	97336	6.7823	3.5830	96	9216	884736	9.7980	4.5789
47 48	2304	103823	6.8557 6.9282	3.6088 3.6342	97 98	9409	912673 941192	9.8489	4.5947
49	2301	117649	7.0000	3.6593	90	9801	941192	0.0400	4.6261
50	-+	125000	7.0711	3.6840	100	10000	1000000	10.0000	4.6416

SQUARES, CUBES, SQUARE AND CUBE ROOTS OF NUMBERS FROM 1 TO 1000

No.	Square	Cube	Sq. Root	Cube Root	No.	Square	Cube	Sq. Root	Cube Root
101	10201	1030301	10.0499	4.6570	151	22801	3442951	12.2882	5.3251
102	10404	1061208	10.0005	4.6723	152	23101	3511808	12.3288	5.3368
103	10600	1092727	10.1489	4.6875	153	23400	3581577	12.3603	5.3485
104	10816	1124864	10.1080	4.7027	154	23716	3652264	12.4097	5.3601
105	11025	1157625	10.2470	4.7177	155	24025	3723875	12.4499	5.3717
106	11236	1101016	10.2056	4.7326	156	24336	3796416	12.4000	5.3832
107	11449	1225043	10.3441	4.7475	157	24649	3869893	12.5300	5-3947
108	11664	1259712	10.3923	4.7622	158	24964	3944312	12.5698	5.4061
100	11881	1205020	10.1403		159	25281	4019679	12.6005	5.4175
110	12100	1331000	10.4881	4.7914	160	25600	4096000	12.6491	5.4288
111	12321	1367631	10.5357	4.8059	161	25021	4173281	12.6886	
112	12544	1404928	10.5830	4.8203	162	26244	4251528	12.7270	5.440I 5.45I4
113	12760	1442897	10.6301	4.8346	163	2656Q	4330747	12.7671	5.4514
114	12006	1481544	10.6771	4.8488	164	26896	4410944	12.8062	5.4737
115	13225	1520875	10.7238	4.8629	165	27225	4492125	12.8452	5.4848
116	13456	1560806	10.7703	4.8770	166	27556	4574296	12.8841	5.4959
117	13689	1601613	10.8167	4.8010	167	27889	4657463	12.0228	5.5069
118	13924	1643032	10.8628	4.9049	168	28224	4741632	12.9615	5.5178
110	14161	1685150	10.9087	4.9187	160	28561	4826800	13.0000	5.5288
120	14400	1728000	10.9545	4.9324	170	28900	4913000	13.0384	5.5397
	6	6-							
I2I I22	14641 14884	1771561	11,0000	4.9461	171	29241	5000211	13.0767	5.5505
123		1815848 1860867	11.0454	4.9597	172	29584	5088448	13.1149	5.5613
124	15129	1906624	11.0905	4.9732 4.9866	173 174	29929 30276	5177717 5268024	13.1529	5.5721
125	15625	1953125	11.1353	5.0000	175	30625	5359375	13.1909	5.5828
126	15876	2000376	11,2250	5.0133	176	30076	5451776	13.2665	5.6041
127	16120	2048383	11,2604	5.0265	177	31329	5545233	13.3041	5.6147
128	16384	2007152	11.3137	5.0397	178	31684	5639752	13.3417	5.6252
120	16641	2146680	11.3578	5.0528	170	32041	5735339	13.3791	5.6357
130	16900	2197000	11,4018	5.0658	180	32400	5832000	13.4164	5.6462
131	17161	2248001		5.0788	181	32761	#000F4F		- 6-6-
132	17424	2200068	11.4455	5.0916	182	33124	5929741 6028568	13.4536	5.6567 5.6671
133	17680	2352637	11.4391	5.1045	183	33489	6128487	13.4907	5.6774
134	17956	2406104	11.5758	5.1172	184	33856	6220504	13.5647	5.6877
135	18225	2460375	11.6190	5.1299	185	34225	6331625	13.6015	5.6980
136	18496	2515456	11.6610	5.1426	186	34596	6434856	13.6382	5.7083
137	18769	2571353	11.7047	5.1551	187	34969	6539203	13.6748	5.7185
138	19044	2628072	11.7473	5.1676	188	35344	6644672	13.7113	5.7287
130	19321	2685610	11.7808	5.1801	180	35721	6751260	13.7477	5.7388
140	19600	2744000	11.8322	5.1925	190	36100	6859000	13.7840	5.7489
141	19881	2803221	11.8743	5.2048	101	36481	6967871	13.8203	£ 7500
142	20164	2863288	11.0164	5.2171	192	36864	7077888	13.8564	5.7590
143	20104	2003200	11.9583	5.2293	193	37249	7189057	13.8924	5.7790
144	20736	2985984	12.0000	5.2415	193	37636	7301384	13.0284	5.7890
145	21025	3048625	12.0416	5.2536	195	38025	7414875	13.9642	5.7989
146	21316	3112136	12.0830	5.2656	106	38416	7529536	14.0000	5.8088
147	21600	3176523	12.1244	5.2776	197	38809	7645373	14.0357	5.8186
148	21904	3241792	12.1655	5.2896	198	39204	7762392	14.0712	5.8285
149	22201	3307949	12.2066	5.3015	199	3960I	7880599	14.1067	5.8383
150	22500	3375000	12.2474	5.3133	200	40000	8000000	14.1421	5.8480
	i				1				

Squares, Cubes, Square and Cube Roots of Numbers from 1 to 1000

No.	Square	Cube	Sq. Root	Cube Root	No.	Square	Cube	Sq. Root	Cube Root
201 202 203 204 205 206 207 208 209 210	40401 40804 41209 41616 42025 42436 42849 43264 43681 44100	8120601 8242408 8365427 8489664 8615125 8741816 8869743 8998912 9120329 9261000	14.1774 14.2127 14.2478 14.2829 14.3178 14.3527 14.3875 14.4222 14.4568 14.4914	5.8578 5.8675 5.8771 5.8868 5.8964 5.9059 5.9155 5.9250 5.9345 5.9439	251 252 253 254 255 256 257 258 259 260	63001 63504 64009 64516 65025 65536 66049 66564 67081 67600	15813251 16003008 16194277 16387064 16581375 16777216 16974593 17173512 17373979 17576000	15.8430 15.8745 15.9060 15.9374 15.9687 16.0000 16.0312 16.0624 16.0935 16.1245	6,3080 6,3164 6,3247 6,3330 6,3413 6,3496 6,3579 6,3661 •6,3743 6,3825
211 212 213 214 215 216 217 218 219 220		9393931 9528128 9663597 9800344 9938375 10077696 10218313 10360232 10503459 10648000	14.5258 14.5602 14.5945 14.6287 14.6629 14.7696 14.7309 14.7648 14.7986 14.8324	5.9533 5.9627 5.9721 5.9814 5.9907 6.0000 6.0092 6.0185 6.0277 6.0368	261 262 263 264 265 266 267 268 269 270	68121 68644 69169 69696 70225 70756 71289 71824 72361 72900	17779581 17984728 18191447 18399744 18609625 18821096 19034163 19248832 19465109 19683000	16.1555 16.1864 16.2173 16.2481 16.2788 16.3095 16.3401 16.3707 16.4012 16.4317	6.3907 6.3988 6.4070 6.4151 6.4232 6.4312 6.4393 6.4473 6.4553 6.4633
22I 222 223 224 225 226 227 228 229 230	48841 49284 49729 50176 50625 51076 51529 51984 52441 52900	10793861 10941048 11089567 11239424 11390625 11543176 11697083 11852352 12008989 12167000	14.8661 14.8997 14.9332 14.9666 15.0000 15.0333 15.0665 15.0997 15.1327 15.1658	6.0459 6.0550 6.0641 6.0732 6.0822 6.0912 6.1002 6.1001 6.1180 6.1269	271 272 273 274 275 276 277 278 279 280	73441 73984 74529 75076 75625 76176 76729 77284 77841 78400	19902511 20123648 20346417 20570824 20790875 21024576 21253933 21484952 21717639 21952000	16.4621 16.4924 16.5227 16.5529 16.5831 16.6132 16.6433 16.733 16.733	6.4713 6.4792 6.4872 6.4951 6.5030 6.5108 6.5187 6.5265 6.5343 6.5421
231 232 233 234 235 236 237 238 239 240	53361 53824 54289 54756 55225 55696 56169 56644 57121 57600	12326391 12487168 12649337 12812904 12977875 13144256 13312053 13481272 13651919 13824000	15.1987 15.2315 15.2643 15.2971 15.3297 15.3623 15.3948 15.4272 15.4596 15.4919	6.1358 6.1446 6.1534 6.1622 6.1710 6.1797 6.1885 6.1972 6.2058 6.2145	281 282 283 284 285 286 287 288 289 290	78961 79524 80089 80656 81225 81796 82369 82944 83521 84100	22188041 22425768 22665187 22906304 23149125 2339365 23639903 23887872 24137569 24389000	16.7631 16.7929 16.8226 16.8523 16.8819 16.9115 16.9411 16.9706 17.0000 17.0294	6.5499 6.5577 6.5654 6.5731 6.5808 6.5885 6.5962 6.6039 6.6115 6.6191
241 242 243 244 245 246 247 248 249 250	61009 61504 62001	13997521 14172488 14348907 14526784 14706125 14886936 15069223 15252992 15438249 15625000	15.5242 15.5563 15.5885 15.6205 15.6525 15.6844 15.7162 15.7480 15.7797 15.8114	6.2231 6.2317 6.2403 6.2488 6.2573 6.2658 6.2743 6.2828 6.2912 6.2996	291 292 293 294 295 296 297 298 200 300	84681 85264 85849 86436 87025 87616 88209 88804 89401 90000	24642171 24897088 25153757 25412184 25672375 25934336 26198073 26463592 26730899 27000000	17.0587 17.0880 17.1172 17.1464 17.1756 17.2047 17.2337 17.2627 17.2916 17.3205	6.6267 6.6343 6.6419 6.6494 6.6569 6.6644 6.6719 6.6794 6.6869 6.6943

SQUARES, CUBES, SQUARE AND CUBE ROOTS OF NUMBERS FROM 1 TO 1000

No.	Square	Cube	Sq. Root	Cube Root	No.	Square	Cube	Sq. Root	Cube Root
301 302 303 304 305 306 307 308 309 310	91809 92416 93025 93636 94249 94864 95481	27270901 27543608 27818127 28094464 28372625 28652616 28934443 29218112 29503629 29791000	17.3494 17.3781 17.4069 17.4356 17.4642 17.4029 17.5214 17.5499 17.5784 17.6068	6.7018 6.7092 6.7166 6.7240 6.7313 6.7387 6.7460 6.7533 6.7606 6.7679	351 352 353 354 355 356 357 358 359 360	123201 123904 124509 125316 126025 126736 127449 128164 128881 129600	43243551 43614208 43986977 44361864 44738875 45118016 45490293 45882712 46268279 46656000	18.7350 18.7617 18.7883 18.8149 18.8414 18.8680 18.8944 18.9209 18.9473 18.9737	7.0540 7.0607 7.0674 7.0740 7.0807 7.0873 7.0940 7.1006 7.1072 7.1138
311 312 313 314 315 317 318 319 320	97344 97969 98596 99225 99856 100489 101124 101761	30080231 30371328 30064297 30959144 31255875 31554496 31855013 32157432 32461759 32768000	17.6352 17.6635 17.6918 17.7200 17.7482 17.7764 17.8045 17.8326 17.8606 17.8885	6.7752 6.7824 6.7897 6.7969 6.8041 6.8113 6.8185 6.8256 6.8323 6.8399	361 362 363 364 365 366 367 368 369 370	130321 131044 131769 132496 133225 133956 134689 135424 136161	48627125 49027896 49430863	19.0000 19.0263 19.0526 19.0788 19.1050 19.1311 19.1572 19.1833 19.2094 19.2354	7.1204 7.1269 7.1335 7.1400 7.1466 7.1531 7.1596 7.1661 7.1726 7.1791
321 322 323 324 325 327 328 329 339	103684 104329 104976 105625 106276 106929 107584 108241	33076161 33386248 33698267 34012224 34328125 34645576 34965783 35287552 35611289 35937000	17.9165 17.9444 17.9722 18.0000 18.0278 18.0555 18.0831 18.1108 18.1384 18.1659	6.8470 6.8541 6.8612 6.8683 6.8753 6.8824 6.8894 6.8964 6.9034 6.9104	371 372 373 374 375 376 377 378 379 380	137641 138384 139129 139876 140625 141376 142129 142884 143641 144400	51895117 52313624 52734375 53157376 53582633 54010152 54439939	19.3391 19.3649 19.3907 19.4165 19.4422 19.4679	7.1855 7.1920 7.1984 7.2048 7.2112 7.2177 7.2240 7.2304 7.2368 7.2432
331 332 333 334 335 337 338 339	110224 110889 111556 112225 112896 113569 114244 114921	36264691 36594368 36926037 37259704 37595375 37933056 38272753 38614472 38958219 39304000	18.1934 18.2209 18.2483 18.2757 18.3030 18.3303 18.3576 18.3848 18.4120 18.4391	6.9174 6.9244 6.9313 6.9382 6.9451 6.9521 6.9589 6.9658 6.9727 6.9795	381 382 383 384 385 386 387 388 389 390	145161 145024 146689 147456 148225 148996 149769 150544 151321 152100	56181887 56623104 57066623 57512456 57060603 58411072 58863869	19.6977 19.7231	7.2622 7.2685 7.2748
341 343 344 345 347 347 347 347 347 347	116964 117649 1118336 119025 119716 120409 121104	40353607 40707584 41063625 41421736 41781923 42144192 42508549		6.9864 6.9932 7.0000 7.0068 7.0136 7.0203 7.0271 7.0338 7.0406 7.0473	391 392 393 394 395 396 397 398 309 400	152881 153664 154449 155236 156025 156816 157609 158404 159201 160000	60698457 61162984 61629875 62099136 62570773 63044792 63521199	19.8242 19.8494 19.8746 19.8997 19.9249 19.9499 19.9750	7.3248 7.3310 7.3372 7.3434 7.3406 7.3558 7.3619

Squares, Cubes, Square and Cube Roots of Numbers from 1 to 1000

No.	Square	Cube	Sq. Root	Cube Root	No.	Square	Cube	Sq. Root	Cube Root
401 402 403 404 405 406 407 408	163216 164025 164836 165649 166464	64481201 64964808 65450827 65939264 66430125 66923416 67419143 67917312 68417929	20.0250 20.0499 20.0749 20.0998 20.1246 20.1494 20.1742 20.1990 20.2237	7.3742 7.3803 7.3864 7.3925 7.3986 7.4047 7.4108 7.4169 7.4229	451 452 453 454 455 456 457 458 459	203401 204304 205209 206116 207025 207936 208849 209764 210681	-91733851 92345408 92959677 93576664 94196375 94818816 95443993 96071912	21.2603 21.2838 21.3073 21.3307 21.3542 21.3776 21.4009	7.6688 7.6744 7.6800 7.6857 7.6914 7.6970 7.7026 7.7082 7.7138
410			20.2485	7.4290	460	211600	97336000		7.7194
411 412 413 414 415 416 417 418 419 420	170569 171396 172225 173056 173889 174724 175561	69426531 69934528 70444997 70957944 71473375 71991296 72511713 73034632 73560059 74088000	20.2731 20.2078 20.3224 20.3470 20.3715 20.3961 20.4206 20.4450 20.4695 20.4939	7.4350 7.4410 7.4470 7.4530 7.4590 7.4650 7.4710 7.4770 7.4829 7.4889	461 462 463 464 465 466 467 468 469 470	212521 213444 214369 215296 216225 217156 218089 219024 219961 220900	97972181 98611128 99252847 99897344 100544625 101194696 101847563 102503232 103161709	21.4942 21.5174 21.5407 21.5639 21.5870 21.6102 21.6333 21.6564	7.7250 7.7306 7.7362 7.7418 7.7473 7.7529 7.7684 7.7639 7.7695 7.7750
421 422 423 424 425 426 427 428 429 430	178084 178929 179776 180625 181476 182329 183184 184041	74618461 75151448 75686967 76225024 76765625 77308776 77854483 78402752 78953589 79507000	20.5183 20.5426 20.5670 20.5913 20.6155 20.6398 20.6640 20.6882 20.7123 20.7364	7.4948 7.5007 7.5067 7.5126 7.5185 7.5244 7.5302 7.5361 7.5420 7.5478	47I 472 473 474 475 476 477 478 479 480	221841 222784 223729 224676 225625 226576 227529 228484 229441 230400	105154048 105823817 106496424 107171875 107850176 108531333 109215352	21.7256 21.7486 21.7715 21.7945 21.8174 21.8403 21.8632 21.8861	7.7805 7.7860 7.7915 7.7970 7.8025 7.8079 7.8134 7.8188 7.8243 7.8297
431 432 433 434 435 436 437 438 439	186624 187489 188356 189225 190096 190969 191844 192721	80062991 80621568 81182737 81746504 82312875 82881856 83453453 84027672 84604519 85184000	20.7605 20.7846 20.8087 20.8327 20.8567 20.8806 20.9045 20.9284 20.9523 20.9762	7-5537 7-5595 7-5654 7-5712 7-5770 7-5828 7-5886 7-5944 7-6001 7-6059	481 482 483 484 485 486 487 488 489 490	231361 232324 233289 234256 235225 236196 237169 238144 239121 240100	111980168 112678587 113379904 114084125 114791256 115501303 116214272	21.9545 21.9773 22.0000 22.0227 22.0454 22.0681 22.0907 22.1133	7.8352 7.8406 7.8460 7.8514 7.8568 7.8622 7.8676 7.8730 7.8784 7.8837
441 442 443 444 445 446 447 448 449	195364 196249 197136 198025 198916 199809 200704 201601	85766121 86350888 86938307 87528384 88121125 88716536 89314623 89915392 90518849 91125000	21.0000 21.0238 21.0476 21.0713 21.0950 21.1187 21.1424 21.1660 21.1896 21.2132	7.6117 7.6174 7.6232 7.6289 7.6346 7.6403 7.6460 7.6517 7.6574 7.6631	491 492 493 494 495 496 497 498 499 500	241081 242064 243049 244036 245025 246016 247009 248004 249001 250000	119823157 120553784 121287375 122023936 122763473 123505992 124251499	22.1811 22.2036 22.2261 22.2486 22.2711 22.2935 22.3159 22.3383	7.8891 7.8944 7.8998 7.9051 7.9155 7.9158. 7.9211 7.9264 7.9317 7.9370

Squares, Cubes, Square and Cube Roots of Numbers from 1 to 1000

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No.	Square	Cube	Sq. Root	Cube Root	No.	Square	Cube	Sq. Root	Cube Root
501	251001	125751501	22.3830	7.9423	551	303601	167284151	23.473.1	8.1982
502	252004	126506008		7.9476	552	304704	168196608	23.4947	8.2031
503	253000	127263527		7.9528	553	305800	169112377		8.2081
0.1	254016	128024064		7.9581	554	306916	170031464		
0.5	255025	128787625		7.9634	555	308025	170053875		8.218
506	256036	120554216		7.9686	556	309136	171879616		8.2220
07	257049	130323843		7.9739	557	310249	172808693	23.6008	8.2278
508	258064	131006512	22.5389	7.9791	558	311364	173741112	23.6220	8.232
00	259081	131872229	22.5610	7.9843	559	312481	174676879	23.6432	8.2377
510	260100	132651000	22.5832	7.9896	560	313600	175616000	23.6643	8.2426
511	261121	133432831	22.6053	7.9948	561	314721	176558481		8.2475
512	262144	134217728	22.6274	8.0000	562	315844	177504328	23.7065	8.2522
513	263169	135005697		8.0052	563	316969	178453547	23.7276	8.2573
14	264196	135796744		8.0104	564	318096		23.7487	8.2621
15	265225	136590875		8.0156	565	319225		23.7697	8.2670
;16	266256	137388096		8.0208	566	320356	181321496		8.2710
517	267289	138188413		8.0260	567	321489	182284263		8.2768
;18	268324	138991832		8.0311	568	322624	183250432		8.2816
19	269361	139798359		8.0363	569	323761	184220009		8.2865
20	270400	140608000	22.8035	8.0415	570	324900	185192000	23.8747	8.2913
21	271441	141420761		8.0466	571	326041		23.8956	8.2962
522	272484	142236648		8.0517	572	327184	187149248		8.3010
323	273529	143055667		8.0569	573	328329		23.9374	8.3059
524	274576	143877824		8.0620	574	329476	189119224		8.3107
525	275625	144703125		8.0671	575	330625	190109375		8.3155
;26	276676	145531576		8.0723	576	.331776		24.0000	8.3203
27	277729	146363183		8.0774	577	332929	192100033		8.3251
528	278784 279841	147197952	22.9703	8.0825 8.0876	578	334084 33524I	193100552		8.3348
29 30	280900	148877000		8.0927	579 580	336400	195112000		8.3396
	281061		22.0424	8.0078	581	227561	106122041	24 7020	8.3443
531	283024	149721291		8.1028	582	337561	107137368		8.3491
32	284089	151419437		8.1070	583	339889	198155287		8.3530
533 534	285156	152273304		8.1130	584	341056	100176704		8.3587
535	286225	153130375		8.1180	585	342225	200201625		8.362
536	287296	153990656	23.1517	8.1231	586	343396	201230056		8.3682
37	288369	154854153		8.1281	587	344569	202262003		8.3730
38	289444	155720872	23.1018	8.1332	588	345744	203297472		8.3777
39	200521	156590819	23.2164	8.1382	589	346921	204336469		8.3825
40	291600	157464000		8.1433	590	348100	205379000	24.2899	8.3872
541	292681	158340421	23.2504	8.1483	591	349281	206425071	24.3105	8.3919
542	293764	150220088		8.1533	592	350464	207474688		8.3967
543	294849	160103007		8.1583	593	351649	208527857		8.401
544	295936	160989184		8.1633	594	352836	209584584		8.4061
545	297025	161878625		8.1683	595	354025		24.3926	8.4108
546	298116	162771336		8.1733	596	355216	211708736	24.4131	8.415
547	299209	163667323		8.1783	597	356409		24.4336	8.4202
		164566592		8.1833	598	357604	213847192	24.4540	8.4249
	300304	1045003921	23.40941						
548 549	300304	165469149		8.1882	599 600	358801 360000	214921799		8.4296

Squares, Cubes, Square and Cube Roots of Numbers from 1 to 1000

No.	Square	Cube	Sq. Root	Cube Root	No.	Square	Cube	Sq. Root	Cube Root
601	361201	217081801	24.5153	8.4390	651	423801	275894451		8.6668
602	362404	218167208		8.4437	652	425104	277167808		8.6713
603	363609	219256227 220348864		8.4484 8.4530	654	426409 427716	278445077 279726264		8.6757 8.6801
605	364816	221445125		8.4577	655	429025	281011375		8.6845
606	367236	222545016	21.6171	8,4623	656	430336	282300416	25.6125	8.6800
607	368449			8.4670	657	431649	283593393	25.6320	8.6934
608	369664	224755712		8.4716	658	432964	284890312		8.6978
600	370881	225866529		8.4763	659	434281	286191179	25.6710	8.7022
610	372100	226981000	24,6982	8.4809	660	435600	287496000	25.6905	8.7066
611	373321	228099131		8.4856	661	436921	288804781		8.7110
612	374544	229220928		8.4902	662	438244	290117528		8.7154
613		230346397		8.4948	663	439569			
614	376996	231475544 232608375		8.4994 8.5040	664	442225	292754944 294079625	25.7052	8.724 1 8.7285
615 616	378225 379456	232000375		8.5086	666	443556	295408296		8.7329
617	380689	234885113		8.5132	667	444889	296740963		8.7373
618	381924	236029032	21.8506	8.5178	668	146224	298077632		8.7416
619	383161	237176659		8.5224	669	447561			8.746 o
620	384400	238328000	24.8998	8.5270	670	448900	300763000	25.8844	8.7503
621	385641	230483061	24.9199	8.5316	671	450241	302111711	25.9037	8.7547
622		240641848	24.9399	8.5362	672	451584	303464448	25.9230	8.7590
623		241804367		8.5408	673	452929	304821217		8.7634
624		242970624		8.5453	674	454276	306182024		8.7677
625		244140625		8.5499	675	455625	307546875		8.7721
626		245314376 246491883		8.5544 8.5590	676	456976	308915776 310288733	26.0000	8.7764 8.7807
627 628		247673152		8.5635	678	459684	311665752		8.7850
620		248858189		8.5681	679	461041	313046839		8.7893
630		250047000		8.5726	680	462400	314432000	26.0768	8.7937
631	398161	251239591	25.1197	8.5772	681	463761	315821241	26.0960	8.7980
632		252435968		8.5817	682	465124	317214568		
633		253636137		8.5862	683	466489	318611987		8.8066
634				8.5907	684	467856	320013504		
635		256047875	25.1992	8.5952	685	469225	321419125		
636		257259456 258474853	25.2190	8.5997 8.6043	687	470596	322828856		8.8237
637 638			25.2587	8.6088	688	473344	325660672		
639		260017110		8.6132	689	474721	327082769		
640		262144000		8.6177	690	476100	328509000		
641	410881	263374721	25.3180	8.6222	601	477481	329939371	26.2869	8.8408
642				8.6267	692	478864	331373888		8.8451
643				8.6312	693	480249	332812557	26.3249	8.8493
644	414736	267080084	25.3772	8.6357	694	481636	334255384	26.3439	8.8536
645	416025	268336125		8.6401	695	483025	335702375		
646				8.6446	696	484416	337153536	20.3818	8.8621
647	418609			8.6490	697	485809	338608873		8.8663 8.8706
648 649				8.6535 8.6579	698	487204 488601	340068392		
650					700	400000	343000000		
0,0	422,00	-,402,000	-3.4934	1 0.0004	11	1,922.00	1 54511100	1.4373	

SQUARES, CUBES, SQUARE AND CUBE ROOTS OF NUMBERS FROM
1 TO 1000

No.	Square	Cube	Sq. Root	Cube Root	No.	Square	Cube	Sq. Root	Cube Root
701 702 703 704 705 706 707 708 709 710	491401 492804 494209 495616 497025 498436 409849 501264 502681 504100	344472101 345048408 347428027 348913664 350402625 351895816 353303243 354894912 356400829 357911000	26.4953 26.5141 26.5330 26.5518 26.5707 26.5895 26.6083 26.6271	8.8833 8.8875 8.8917 8.8959 8.9001 8.9043 8.9085 8.9127 8.9169 8.9211	751 752 753 754 755 756 757 758 759 760	564001 565504 567009 568516 570025 571536 573049 574564 576081 577600	423564751 425259008 426957777 428661064 430368875 432081216 433798093 435519512 437245470 438976000	27.4226 27.4408 27.4591 27.4773 27.4955 27.5136 27.5318 27.5500	9.0896 9.0937 9.0977 9.1017 9.1057 9.1098 9.1138 9.1178 9.1218 9.1258
711 712 713 714 715 716 717 718 719 720	505521 506944 508369 509796 511225 512656 514089 515524 516961 518400	359,425431 3609,44128 362,467097 363,9943,44 365525875 367061696 368601813 3701,46232 37169,4959 37324,8000	26.6833 26.7021 26.7208 26.7395 26.7582 26.7769 26.7955 26.8142	8.9253 8.9295 8.9337 8.9378 8.9420 8.9462 8.9503 8.9545 8.9587 8.9628	761 762 763 764 765 766 767 768 769 770	579121 580644 582169 583696 585225 586756 588289 589824 591361 592900	440711081 442450728 444194947 445943744 447697125 449455096 451217663 452984832 454756609 456533000	27.6043 27.6225 27.6405 27.6586 27.6767 27.6948 27.7128 27.7308	9.1298 9.1338 9.1378 9.1418 9.1458 9.1498 9.1537 9.1577 9.1617
72I 722 723 724 725 726 727 728 729 730	519841 521284 522729 524176 525625 527076 528529 529984 531441 532900	374805361 376367048 377933067 379503424 381078125 3826577176 38424058 385828352 387420489 389017000	26.8701 26.8887 26.9072 26.9258 26.9444 26.9629 26.9815 27.0000	8.9670 8.9711 8.9752 8.9794 8.9835 8.9836 8.9918 8.9959 9.0000 9.0041	771 772 773 774 775 776 777 778 779 780	594441 595984 597529 599076 600625 602176 603729 605284 606841 608400	458314011 460099648 461889917 463684824 465484375 467288576 469097433 470910952 472729139 474552000	27.7849 27.8029 27.8209 27.8388 27.8568 27.8747 27.8927 27.9106	9.1696 9.1736 9.1775 9.1815 9.1855 9.1894 9.1933 9.1973 9.2012 9.2052
731 732 733 734 735 736 737 738 739 740		390617891 392223168 393832837 395446904 397065375 398688256 400315553 401947272 403583419 405224000	27.0555 27.0740 27.0924 27.1109 27.1293 27.1477 27.1662 27.1846	9.0082 9.0123 9.0164 9.0205 9.0246 9.0287 9.0328 9.0369 9.0410 9.0450	781 782 783 784 785 786 787 788 789	609961 611524 613089 614656 616225 617796 619369 620944 622521 624100	476379541 478211768 480048687 481890304 483736625 485587656 487443403 489303872 491169069 493039000	27.9643 27.9821 28.0000 28.0179 28.0357 28.0535 28.0713 28.0891	9.2091 9.2130 9.2170 9.2209 9.2248 9.2287 9.2326 9.2365 9.2404 9.2443
741 742 743 744 745 746 747 748 749	552049 553536 555025 556516 558009 559504 561001	413493625 415160936 416832723 418508992 420189749	27.2397 27.2580 27.2764 27.2947 27.3130 27.3313 27.3496 27.3679	9.0491 9.0532 9.0572 9.0673 9.0654 9.0694 9.0735 9.0775 9.0816 9.0856	791 792 703 794 795 796 797 798 799 800	625681 627264 628849 630436 632025 633616 635209 636804 638401 640000	494913671 49679388 498677257 500566184 502459875 504358336 506261573 508160592 510082399 512000000	28.1425 28.1603 28.1780 28.1957 28.2135 28.2312 28.2489 28.2666	9.2482 9.2521 9.2560 9.2599 9.2638 9.2677 9.2716 9.2754 9.2793 9.2832

Squares, Cubes, Square and Cube Roots of Numbers from 1 to 1000

No.	Square	Cube	Sq. Root	Cube Root	No.	Square	Cube	Sq. Root	Cube Root
801 802 803 804 805 806 807 808 809 810	641601 643204 644809 646416 648025 649636 651249 652864 651481 656100	513922401 515849608 517781627 519718464 521660125 523606616 525557943 527514112 529475129 531441000	28.3196 28.3373 28.3549 28.3725 28.3901 28.4077 28.4253 28-4429	9.2870 9.2909 9.2948 9.2986 9.3025 9.3063 9.3102 9.3140 9.3179 9.3217	851 852 853 854 855 856 857 858 859 860	724201 725004 727600 729316 731025 732736 734449 736164 737881 739600	616295051 618470208 620650477 622835864 625026375 627222016 629422793 631628712 633839770 636056000	29.1890 29.2062 29.2233 29.2404 29.2575 29.2746 29.2916 29.3087	9.4764 9.4801 9.4838 9.4875 9.4912 9.4940 9.5023 9.5060 9.5097
811 812 813 814 815 816 817 818 819	657721 659344 660969 662596 664225 665856 667489 669124 670761 672400	533411731 535387328 537367797 539353144	28.4781 28.4956 28.5132 28.5330 28.5482 28.5657 28.5657 28.6007 28.6182	9.3255 9.3294 9.3332 9.3370 9.3408 9.3447 9.3485 9.3523 9.3561 9.3599	861 862 863 864 865 866 867 868 869 870	741321 743044 744769 746496 748225 749956 751689 753424 755161 756900	638277381 640503928 642735647 644072544 647214625 649461896 651714363 653972032 656234909 658503000	29.3428 29.3598 29.3769 29.3939 29.4109 29.4279 29.4449 29.4618 29.4788	9.5134 9.5171 9.5207 9.5244 9.5281 9.5317 9.5354 9.5391 9.5427 9.5464
821 822 823 824 825 826 827 828 829 830	674041 675684 677329 678976 680625 682276 683929 685584 687241 688900	553387661 555412248 557441767 559476224 561515625 563559976 565609283 567663552 569722789 571787000	28.6705 28.6880 28.7054 28.7228 28.7402 28.7576 28.7750 28.7924	9.3637 9.3675 9.3713 9.3751 9.3789 9.3827 9.3865 9.3902 9.3940 9.3978	871 872 873 874 875 876 877 878 879 880	758641 760384 762129 763876 765625 767376 769129 770884 772641 774400	660776311 663054848 665338617 667627624 669921875 672221376 674526133 676836152 679151439 681472000	29.5296 29.5466 29.5635 29.5804 29.5973 29.6142 29.6311 29.6479	9.5501 9.5537 9.5574 9.5610 9.5647 9.5683 9.5719 9.5756 9.5792 9.5828
831 832 833 834 835 836 837 838 839 840	690561 692224 693889 695556 697225 698896 700569 702244 703921 705600	573856191 575930368 578009537 580093704 582182875 584277056 586376253 588480472 590589719 592704000	28.8444 28.8617 28.8791 28.8964 28.9137 28.9310 28.9482 28.9655	9.4016 9.4053 9.4091 9.4129 9.4166 9.4204 9.4241 9.4279 9.4316 9.4354	881 882 883 884 885 886 887 888 889 890	776161 777924 779689 781456 783225 784996 786769 788544 790321 792100	683797841 686128968 688465387 690807104 693154123 695506456 697864103 700227072 702595360 704969000	29.6985 29.7153 29.7321 29.7489 29.7658 29.7825 29.7993 29.8161	9.5865 9.5901 9.5937 9.5973 9.6010 9.6046 9.6082 9.6118 9.6154 9.6190
841 842 843 844 845 846 847 848 849 850	720801	594823321 596947688 599077107 601211584 603351125 605495736 607645423 609800192 611960049 614125000	29.0172 29.0345 29.0517 29.0689 29.0861 29.1033 29.1204 29.1376	9.4391 9.4429 9.4466 9.4503 9.4541 9.4578 9.4615 9.4652 9.4690 9.4727	891 892 893 894 895 896 897 898 899 900	793881 795664 797449 799236 801025 802816 804609 806404 808201 810000	707347971 709732288 712121957 714516984 716917375 719323136 721734273 724150792 726572699 729000000	29.866.1 29.8831 29.8998 29.9166 29.9333 29.9500 29.9666 29.9833	9.6226 9.6262 9.6298 9.6334 9.6370 9.6406 9.6442 9.6477 9.6513 9.6549

Squares, Cubes, Square and Cube Roots of Numbers from 1 TO 1000

No.	Square	Cube	Sq. Root	Cube Root	No.	Square	Cube	Sq. Root	Cube Root
901 902 903 904 905 906 907 908 909	811801 813604 815409 817216 819025 820836 822649 824464 826281 828100	746142643 748613312	30.0333 30.0500 30.0666 30.0832 30.0998 30.1164 30.1330	9.6585 9.6620 9.6656 9.6692 9.6727 9.6763 9.6799 9.6834 9.6870 9.6905	951 952 953 954 955 956 957 958 959 960	904.401 906304 908209 910116 912025 913936 915849 917764 919681 921600	860085351 862801408 865523177 868250664 870983875 873722816 876467493 879217912 \$81974079 884736000	30.8545 30.8767 30.8869 30.9031 30.9192 30.9354 30.9516 30.9677	9.8339 9.8374 9.8408 9.8443 9.8477 9.8511 9.8546 9.8580 9.8614 9.8648
911 912 913 914 915 916 917 918 919	829921 831744 833569 835396 837225 839056 840889 842724 844561 846400	761048497	30.1993 30.2159 30.2324 30.2490 30.2655 30.2820 30.2985 30.3150	9.6941 9.6976 9.7012 9.7047 9.7082 9.7118 9.7153 9.7188 9.7224 9.7259	961 962 963 964 965 966 967 968 969	923521 925444 927369 929296 931225 933156 935089 937024 938961 940900	895841344 898632125 901428696 904231063 907039232 909853209	31.0161 31.0322 31.0483 31.0644 31.0805 31.0966 31.1127 31.1288	9.8683 9.8717 9.8751 9.8785 9.8819 9.8854 9.8888 9.8922 9.8956 9.8990
921 922 923 924 925 926 927 928 929 930	848241 850084 851929 853776 855625 857476 859329 861184 863041 864900	786330467 788889024 791453125 794022776 796597983 799178752 801765089	30.3645 30.3809 30.3974 30.4138 30.4302 30.4467 30.4631 30.4795	9.7294 9.7329 9.7364 9.7400 9.7435 9.7470 9.7505 9.7540 9.7575 9.7610	971 972 973 974 975 976 977 978 979 980	942841 944784 946729 948676 950625 952576 954529 956484 958441 960400	915498611 918330048 921167317 924010424 926859375 929714176 932574833 935441352 938313739 941192000	31.1769 31.1929 31.2090 31.2250 31.2410 31.2570 31.2730 31.2890	9.9092 9.9126 9.9160 9.9194 9.9227
931 932 933 934 935 936 937 938 939 940	866761 868624 870489 872356 874225 876096 877969 879844 881721 883600	817400375 820025856	30.5287 30.5450 30.5614 30.5778 30.5941 30.6105 30.6268 30.6431	9.7645 9.7680 9.7715 9.7750 9.7785 9.7819 9.7854 9.7889 9.7924 9.7959	981 982 983 984 985 986 987 988 989	962361 964324 966289 968256 970225 972196 974169 976144 978121 980100	952763904 955671625 958585256 961504803	31.3369 31.3528 31.3688 31.3847 31.4006 31.4166 31.4325 31.4484	9.9363 9.9396 9.9430 9.9464 9.9497 9.9531 9.9565 9.9598 9.9632 9.9666
941 942 943 944 945 946 947 948 949	885481 887364 889249 891136 893025 894916 896809 898704 900601 902500	843908625 846590536	30.6920 30.7083 30.7246 30.7409 30.7571 30.7734 30.7896 30.8058	9.7993 9.8028 9.8063 9.8097 9.8132 9.8167 9.8201 9.8236 9.8270 9.8305	991 992 993 994 995 996 997 998 999	982081 984064 986049 988036 990025 992016 994009 996004 998001	973242271 976191488 979146657 982107784 985074875 988047936 991026973 997002999	31.4960 31.5119 31.5278 31.5436 31.5595 31.5753 31.5911 31.6070	9.9699 9.9733 9.9766 9.9800 9.9833 9.9866 9.9900 9.9933 9.9967

AREAS AND CIRCUMFERENCES OF CIRCLES FROM I TO 100

Dia.	Area	Circum.	Dia.	Area	Circum.	Dia.	Area	Circum.
$\frac{1}{32}$	0.00077	0.098175	2	3.1416	6.28319	5	19.635	15.7080
$\frac{3}{6}$	0.00173	0.147262	16	3.3410	6.47953	1 1 6	20.129	15.9043
16	0.00307	0.196350	$\frac{1}{8}$ $\frac{3}{16}$	3.5466	6.67588	$\frac{\frac{1}{8}}{\frac{3}{1.6}}$	20.629	16.1007
$\frac{3}{32}$	0.00690	0.294524	$\frac{3}{16}$	3.7583	6.87223	1 6	21.135	16.2970
18	0.01227	0.392699	$\frac{1}{4}$	3.9761	7.06858	$\frac{1}{4}$	21.648	16.4934
3.2	0.01917	0.490874	1.5	4.2000	7.26493	16	22.166	16.6897
3.6	0.02761	0.589049	3 8	4.4301	7.46128	3 8	22.691	16.8861
$\frac{1}{8}$ $\frac{5}{32}$ $\frac{3}{16}$ $\frac{7}{32}$	0.03758	0.687223	$\begin{array}{ c c c }\hline 1\\ 4\\ \hline 1\\ \hline 6\\ \hline \\ 3\\ \hline \\ 7\\ \hline 1\\ \hline 6\\ \end{array}$	4.6664	7.65763	$ \begin{array}{c c} \frac{1}{4} \\ 5 \\ \hline 16 \\ 3 \\ 8 \\ 7 \\ \hline 16 \end{array} $	23.221	17.0824
$\frac{1}{4}$ $\frac{9}{3}$ $\frac{2}{3}$	0.04909	0.785398	1 9 16	4.9087	7.85398	$\begin{array}{c} \frac{1}{2} \\ \frac{9}{16} \\ \frac{5}{8} \\ \frac{11}{16} \\ \frac{3}{4} \end{array}$	23.758	17.2788
$\frac{9}{32}$	0.06213	0.883573	76	5.1572	8.05033	9 1 6	24.301	17.4751
15	0.07670	0.981748	5.	5.4119	8.24668	5 8	24.850	17.6715
11	0.09281	1.07002	58 11 15 34 13 15	5.6727	8.44303	11	25.406	17.8678
3	0.11045	1.17810	$\frac{3}{4}$	5.9396	8.63938	$\frac{3}{4}$	25.967	18.0642
$\frac{1}{3}\frac{3}{2}$	0.12962	1.27627	13	6.2126	8.83573	13	26.535	18.2605
7 7	0.15033	1.37445	$\frac{7}{8}$	6.4918	9.03208	13 16 7 8	27.109	18.4569
5 5 121 21 20 20 20 20 20 20 20 20 20 20 20 20 20	0.17257	1.47262	$\begin{array}{c} \frac{7}{8} \\ \frac{15}{16} \end{array}$	6.7771	9.22843	$\frac{15}{16}$	27.688	18.6532
$\frac{1}{2}$ $\frac{17}{32}$ $\frac{1}{3}$ $\frac{1}{2}$	0.19635	1.57080	3	7.0686	9.42478	6	28.274	18.8496
$\frac{1}{3}\frac{7}{2}$	0.22166	1.66897	1 1 8 8 3 16	7.3662	9.62113	$\left \frac{1}{8} \right $	29.465	19.2423
16	0.24850	1.76715	1/8	7.6699	9.81748	1	30.680	19.6350
19	0.27688	1.86532	3	7.9798	10.0138	3	31.010	20.0277
5 8	0.30680	1.96350	1	8.2958	10.2102	$\frac{1}{2}$	33.183	20.4204
$\frac{21}{32}$	0.33824	2.06167	1 6	8.6179	10.4065	5 8	34.472	20.8131
11	0.37122	2.15984	38	8.9462	10.6029	3	35.785	21.2058
5 8 212 116 232 232	0.40574	2.25802	7	9.2806	10.7992	14300101003470	37.122	21.5984
34	0.44179	2.35619		9.6211	10.9956	7	38.485	21.9911
$\frac{2}{3}\frac{5}{2}$	0.47937	2.45437	1 9 16	9.9678	11.1010	1 1	39.871	22.3838
13	0.51849	2.55254	5	10.321	11.3883	1 1	41.282	22.7765
2 32 3 67 2 1 12 3 7 8	0.55914	2.65072	11	10.680	11.5846	1(81/43)81/215/83/47/8	42.718	23.1692
7 8	0.60132	2.74889	3	11.045	11.7810	1 3	44.179	23.5619
$\frac{29}{32}$	0.64504	2.84707	13	11.416	11.9773	5 8	45.664	23.9546
15	0.69029	2.94524	78	11.793	12.1737	3	47.173	24.3473
2925 156 132 32	0.73708	3.04342	58 116 34 136 18 158 158	12.177	12.3700	7 8	48.707	24.7400
1	0.78540	3.14159	4	12.566	12.5664	8	50.265	25.1327
$\frac{1}{16}$	0.88664	3.33794		12.962	12.7627	1/8	51.849	25.5224
$\begin{array}{c} \frac{1}{8} \\ \frac{3}{16} \end{array}$	0.99402	3.53429	1 8	13.364	12.9591	1	53.456	25.9181
$\frac{3}{16}$	1.1075	3.73064	3	13.772	13.1554	3/8	55.088	26.3108
4	1.2272	3.92699	1 1 6 1 8 3 1 6 1 4 5 1 6 1 6 1 6 1 6 1 6 1 6 1 6 1 6 1 6 1	14.186	13.3518	14381212583478	56.745	26.7035
16	1.3530	4.12334	1.6	14.607	13.5481	5/8	58.426	27.0962
3	1.4849	4.31969	38	15.033	13.7445	3 4	60.132	27.4889
$\frac{\frac{3}{8}}{\frac{7}{16}}$	1.6230	4.51604	1 6 3 8 7 7 7 6	15.466	13.9408	7 8	61.862	27.8816
1296581634367856	1.7671	4.71239	1 3	15.904	14.1372	9	63.617	28.2743
$\frac{9}{16}$	1.9175	4.90874	16	16.349	14.3335	$\frac{1}{8}$	65.397	28.6670
$\frac{5}{8}$	2.0739	5.10509	5 8	16.800	14.5299	1 4	67.201	29.0597
$\frac{11}{16}$	2.2365	5.30144	$\frac{\frac{5}{8}}{\frac{11}{16}}$	17.257	14.7262	3 8	69.c29	29.4524
34	2.4053	5.49779	1 4	17.721	14.9226	$\frac{1}{2}$	70.882	29.8451
$\frac{13}{16}$	2.5802	5.69414	13	18.190	15.1189	5 8	72.760	30.2378
$\frac{7}{8}$	2.7612	5.89049	17 8 15 16	18.665	15.3153	1/20 1/4 3/20 1/315/20 3/4 7/8	74.662	30.6305
$\frac{15}{16}$	2.9483	6.08684	$\frac{15}{16}$	19.147	15.5116	1 7 8	76.589	31.0232

Areas And Circumferences of Circles from 1 To 100

Dia.	Area	Circum.	Dia.	Area	Circum.	Dia.	Area	Circum.
10 18 14 38 12 55 88 34 77 8	78.540 80.516 82.516 84.541 86.590 88.664 90.763 92.886	31.4159 31.8086 32.2013 32.5940 32.9867 33.3794 33.7721 34.1648	16 1/43/801/215/83/41/8	201.06 204.22 207.39 210.60 213.82 217.08 220.35 223.65	50.2655 50.6582 51.0509 51.4436 51.8363 52.2290 52.6217 53.0144	22 18 14 33 8 12 15 8 34 7 7 8	380.13 384.46 388.82 393.20 397.61 402.04 406.49 410.97	69.1150 69.5077 69.9004 70.2931 70.6858 71.0785 71.4712 71.8639
II 18 143881258334778	95.033 97.205 99.402 101.62 103.87 106.14 108.43	34·5575 34·9502 35·3429 35·7356 36·1283 36·5210 36·9137 37·3064	17 18143812583478	226.98 230.33 233.71 237.10 240.53 243.98 247.45 250.95	53.4071 53.7993 54.1925 54.5852 54.9779 55.3706 55.7633 56.1560	23 18 14 38 12 58 34 77 8	415.48 420.00 424.56 429.13 433.74 438.36 443.01 447.69	72.2566 72.6493 73.0420 73.4347 73.8274 74.2201 74.6128 75.0055
12 18 1438125833478	113.10 115.47 117.86 120.28 122.72 125.19 127.68 130.19	37.6991 38.0918 38.4845 38.8772 39.2699 39.6626 40.0553 40.4480	S 1/201/4/3/81/215/83/447/8	254.47 258.02 261.59 265.18 268.80 272.45 276.12 279.81	56.5487 56.9414 57.3341 57.7268 58.1195 58.5122 58.9049 59.2976	2.1	452·39 457·11 461·86 466·64 471·44 476·26 481·11 485·98	75.3982 75.7909 76.1836 76.5765 76.9690 77.3617 77.7544 78.1471
13	132.73 135.30 137.89 140.50 143.14 145.80 148.49 151.20	40.8407 41.2334 41.6261 42.0188 42.4115 42.8042 43.1969 43.5896	1914300100004170	283.53 287.27 291.04 294.83 298.65 302.49 306.35 310.24	59.6903 60.0830 60.4757 60.8684 61.2611 61.6538 62.0465 62.4392	25 18143881225831478	490.87 495.79 500.74 505.71 510.71 515.72 520.77 525.84	78.5398 78.9325 79.3252 79.7179 80.1105 80.5033 80.8960 81.2887
14 18 14 38 12 58 34 78	153.94 156.70 159.48 162.30 165.13 167.99 170.87 173.78	43.9823 44.3750 44.7677 45.1604 45.5531 45.9458 46.3385 46.7312	20 1/8 1/4 3/8 1/215/8 3/4 7/8	314.16 318.10 322.06 326.05 330.06 334.10 338.16 342.25	62.8319 63.2246 63.6173 64.0100 64.4026 64.7953 65.1880 65.5807	26 1814381 2158831478	530.93 536.05 541.19 546.35 551.55 556.76 562.00 567.27	81.6814 82.0741 82.4668 82.8595 83.2522 83.6449 84.0376 84.4303
15 18143813583478	176.71 179.67 182.65 185.66 188.69 191.75 194.83	47.1239 47.5166 47.9093 48.3020 48.6947 49.0874 49.4801 49.8728	2 I 1814 3812 5583 4778	346.36 350.50 354.66 358.84 363.05 367.28 371.54 375.83	65.9734 66.3661 66.7588 67.1515 67.5442 67.9369 68.3296 68.7223	27 181438121518314718	572.56 577.87 583.21 588.57 593.96 599.37 604.81 610.27	84.8230 85.2157 85.6084 86.0011 86.3938 86.7865 87.1792 87.5719

AREAS AND CIRCUMFERENCES OF CIRCLES FROM I TO 100

28				the same of the sa				Circum.
10143010500470	615.75 621.26 626.80 632.36 637.94 643.55 649.18 656.84	87.9646 88.3573 88.7500 89.1427 89.5354 89.9281 90.3208 90.7135	3+ 181(43)861(245)83(447)8	907.92 914.61 921.32 928.06 934.82 941.61 948.42 955.25	108.385 108.788 109.170 109.563	40 1/81/4/3/80 1/21/5/8/3/47/8	1256.6 1264.5 1272.4 1280.3 1288.2 1296.2 1304.2 1312.2	125.664 126.056 126.449 126.842 127.235 127.627 128.020 128.413
0 (0 (4 (4 (0) 0 (4 (660.52 666.23 671.96 677.71 683.49 689.30 695.13 700.98 706.86 712.76 718.69 724.64 730.62	94.6405 95.0332 95.4259 95.8186	35 1/8 1/4 0/8 1/2 1/2 1/2 1/2 1/2 1/2 1/2 1/2 1/2 1/2	962.11 969.00 975.91 982.84 980.80 996.78 1003.8 1010.8 1017.9 1025.0 1032.1 1039.2 1046.3	100.056 110.348 110.741 111.134 111.527 111.010 112.312 112.705 113.097 113.400 113.883 114.275 114.668	4 I 1	1320.3 1328.3 1336.4 1344.5 1352.7 1360.8 1377.2 1385.4 1393.7 1402.0 1410.3 1418.6	128.805 129.198 129.591 129.993 130.376 130.769 131.161 131.554 131.947 132.340 132.732 133.125 133.518
৪০লেখনখন বিধানীত নামান্ত প্ৰথম হৈ এই বিধানীত নামান্ত প্ৰথম হৈ এই বিধানীত নামান্ত প্ৰথম হৈ এই বিধানীত কৰিব হ	736.62 742.64 748.69 754.77 760.87 766.99 773.14 779.31 785.51 791.73 797.98	96.2113 96.6040 96.9967 97.3894 97.7821 98.1748 98.5675 98.9602 99.3529 99.7456 100.138	37 18 14 18 10 10 10 10 10 10 10 10 10 10 10 10 10	1053.5 1060.7 1068.0 1075.2 1082.5 1089.8 1097.1 1104.5 1111.8 1119.2 1126.7	115.061 115.454 115.846 116.239 116.632 117.024 117.417 117.810 118.202 118.596 118.988	43 1/8 1/4/3/8 1/215/8/3/4/7/8	1427.0 1435.4 1443.8 1452.2 1460.7 1469.1 1477.6 1486.2 1494.7 1503.3 1511.9	133.910 134.303 134.696 135.088 135.481 135.874 136.267 136.659 137.052 137.445 137.837
3 1/81/49/81/2016/89/47/8 1/81/49/81/2016/89/47/8	804.25 810.54 816.86 823.21 829.58 835.97 842.39 848.83 855.30 861.79 868.31 874.85 881.41 888.00 894.62 901.26	100.924 101.316 101.709 102.102 102.494 102.887 103.280	39	1134.1 1141.6 1149.1 1156.6 1164.2 1171.7 1179.3 1186.9 1194.6 1202.3 1210.0 1217.7 1225.4 1233.2 1241.0 1241.0 1248.8	119.381 119.773 120.166 120.559 120.951 121.344 121.737 122.129 122.522 122.915 123.308 123.700 124.093 124.486 124.878	44 48 1(43)8 1(215)8 3(41)8 1(8) 1(8) 1(8) 1(8) 1(8) 1(8) 1(8) 1	1520.5 1529.2 1537.9 1546.6 1555.3 1564.0 1572.8 1581.6 1590.4 1599.3 1608.2 1617.0 1626.0 1634.9 1643.9 1643.9	138.230 138.623 139.015 139.408 139.801 140.194 140.586 140.979 141.372 141.764 142.157 142.550 142.942 143.335 143.728

AREAS AND CIRCUMFERENCES OF CIRCLES FROM I TO 100

Dia.	Area	Circum.	Dia.	Area	Circum.	Dia.	Area	Circum.
46	1661.0	144.513	52	2123.7	163.363	58	2642.1	182.212
	1670.9	144.906	3 ² / ₈	2133.9	163.756	1	2653.5	182.605
1/3 1/4 3/8 1/215/8 3/41/8	1680.0	145.299	Î	2144.2	164.148	1/81/43/81/215/83/47/8	2664.9	182.008
38	1689.1	145.691	143012503478	2154.5	164.541	3/8	2676.4	183.300
$\frac{1}{2}$	1698.2	146.084	$\frac{1}{2}$	2164.8	164.934	1/2	2687.8	183.783
5 8	1707.4	146.477	5 8	2175.1	165.326	5 5	2699.3	184.176
4	1716.5	146.869	4 7	2185.4	165.719	4 4	2710.9	184.569
	1725.7	147.262	1	2195.8	166.112	"	2722.4	184.961
47	1734.9	147.655	53,	2206.2	166.504	59	2734.0	185.354
8	1744.2	148.048	8	2216.6 2227.0	167.290	8	2745.6 2757.2	185.747
3	1753.5 1762.7	148.833	3	2237.5	167.683	3	2768.8	186.532
1/8 1/4 (2) 8 1/2 1/2 (5) 8 (3/4 7/8	1772.1	149.226	100 1 14 3000 1 (2000) 5 3 44 1/5	2248.0	168.075	1/80 1/4/3/80 1/24/5/80 3/4/7/8	2780.5	186.925
<u>5</u>	1781.4	149.618	58	2258.5	168.468	$\frac{5}{8}$	2792.2	187.317
3	1790.8	150.011	3 4	2269.1	168.861	$\frac{3}{4}$	2803.9	187.710
	1800.1	120.404	$\frac{7}{8}$	2279.6	169.253	7 8	2815.7	188.103
48	1809.6	150.796	54	2290.2		60	2827.4	188.496
8	1819.0	151.189	8	2300.8	170.039	8	2839.2	188.888
4 3	1828.5	151.582	1 3	2311.5	170.431	4 3	2851.0 2862.0	189.281
1001430012150034750	1837 9 1847.5	151.975 152.367	8	2322.I 2332.8	170.824 171.217	4 × 1 + 3 00 + 12 15 10 20 + 17 15	2874.8	190.066
5	1857.0	152.760	2 5	2343.5	171.600	2 5	2886.6	190.459
3	1866.5	153.153	3	2354.3	172.002	3	2808.6	190.852
7/8	1876.1	153.544	18011438010058314178	2365.0	172.395	78	2910.5	191.244
49	1885.7	153.938	55	2375.8	172.788	61	2922.5	191.637
	1895.4	154.331	1/8	2386.6	173.180	18	2934.5	192.030
14	1905.0	154.723	$\frac{1}{4}$	2397.5	173.573	4	2946.5	192.423
1/01/43/01/21/503/41/0	1914.7	155.116	18143812583478	2408.3	173.966	1,4300 121500 3 4110	2958.5	192.815
5	1924.2	155.509	5	2419.2	174.358	5	2970.6 2982.7	193.208
8 3	1934.2	155.904 156.294	8 3	2430.I 2441.I	174.751 175.144	8	2902.7	193.001
7	1953.7	156.687	7	2452.0	175.536	47	3006.0	194.386
50	1963.5	157.080	56	2463.0	175.929	62	3019.1	194.779
	1973.3	157.472		2474.0	176.322	1 8	3031.3	195.171
1/4	1983.2	157.865	1/4	2485.0	176.715	1	3043.5	195.564
38	1993.1	158.258	38	2496.1	177.107	38	3055.7	195.957
2	2003.0	158.650	2	2507.2	177.500	$\frac{1}{2}$	3068.0	196.350
8 3	2012.9	159.043	8 3	2518.3	177.893	8 3	3080.3	196.742
1/81/43/81/25/83/47/8	2022.8 2032.8	159.436 159.829	HIS HIM 2000 HIS 11000 00 HI I HIS	2529.4 2540.6	178.285 178.678	143812583478	3092.6	197.135
- 1	2042.8		- 1		' '	1 "	3117.2	197.920
51	2042.8	160.221 160.614	57	2551.8 2563.0	179.071 179.463	63	3129.6	197.920
1 4	2062.0	161.007	1	2574.2	179.856	1	3142.0	198.706
38	2073.0	161.399	38	2585.4	180.240	38	3154.5	199.098
$\frac{1}{2}$	2083.1	161.792	$\frac{1}{2}$	2596.7	180.642	$\frac{1}{2}$	3166.9	199.491
5 8	2093.2	162.185	5	2608.0	181.034	58	3179-4	199.884
181(43)81(25)83(47)8	2103.3	162.577	HT 212 F 12 15 15 25 25 14 17 2	2619.4	181.427	4384258347	3191.9	200.277
8	2113.5	162.970	\$1	2630.7	181.820	8	3204.4	200.009

AREAS AND CIRCUMFERENCES OF CIRCLES FROM I TO 100

Dia.	Area	Circum.	Dia.	Area	Circum.	Dia.	Area	Circum.
64 18 14 38 12 58 34 78	3217.0 3229.6 3242.2 3254.8 3267.5 3280.1 3292.8 3305.6	201.062 201.455 201.847 202.240 202.633 203.025 203.418 203.811	70 1/4/3/20 1/21/5/203/41/20	3848.5 3862.2 3876.0 3889.8 3903.6 3917.5 3931.4 3945.3	219.911 220.304 220.697 221.090 221.482 221.875 222.268 222.660	76	4536.5 4551.4 4566.4 4581.3 4596.3 4611.4 4626.4 4641.5	238.761 239.154 239.546 239.939 240.332 240.725 241.117 241.510
65 18 14 38 12 58 34 78	3318.3 3331.1 3343.9 3356.7 3369.6 3382.4 3395.3 3408.2	204.204 204.596 204.989 205.382 205.774 206.167 206.560 206.952	7 I 1/8 1/4 3/8 1/2 5/8 3/4 1/8	3959.2 3973.1 3987.1 4001.1 4015.2 4029.2 4043.3 4057.4	223.053 223.446 223.838 224.231 224.624 225.017 225.409 225.802	77 18 14 38 12 58 83 34 78	4656.6 4671.8 4686.9 4702.1 4717.3 4732.5 4747.8 4763.1	241.903 242.295 242.688 243.081 243.473 243.866 244.259 244.652
66 1814 388 1215 834 78	3421.2 3434.3 3447.2 3460.2 3473.2 3486.3 3499.4 3512.5	207.345 207.738 208.131 208.523 208.916 209.309 209.701 210.094	72 18 14 38 12158 34478	4071.5 4085.7 4099.8 4114.0 4128.2 4142.5 4156.8 4171.1	226.195 226.587 226.930 227.373 227.765 228.158 228.551 228.944	78 1001443001003141710	4778.4 4793.7 4809.0 4824.4 4839.8 4855.2 4870.7 4886.2	245.044 245.437 245.830 246.222 246.615 247.008 247.400 247.793
67	3525.7 3538.8 3552.0 3565.2 3578.5 3591.7 3605.0 3618.3	210.487 210.879 211.272 211.665 212.058 212.450 212.843 213.236	73 18 14 38 12 58 34 78	4185.4 4199.7 4214.1 4228.5 4242.9 4257.4 4271.8 4286.3	229.336 229.729 230.122 230.514 230.907 231.300 231.692 232.085	79 18 14 38 12 58 34 17 8	4901.7 4917.2 4932.7 4948.3 4963.9 4979.5 4995.2 5010.9	248.186 248.579 248.971 249.364 249.757 250.149 250.542 250.935
68	3631.7 3645.0 3658.4 3671.8 3685.3 3698.7 3712.2 3725.7	213.628 214.021 214.414 214.806 215.199 215.592 215.984 216.337	74 18 14 38 12 58 314 78	4300.8 4315.4 4329.9 4344.5 4359.2 4373.8 4388.5 4403.1	232.478 232.871 233.263 233.656 234.049 234.441 234.334 235.227	৪০ ন্ধাল্ড ন্থান্ত লাখান্ত	5026.5 5042.3 5058.0 5073.8 5089.6 5105.4 5121.2 5137.1	251.327 251.720 252.113 252.506 252.898 253.291 253.684 254.076
69 1814438812258834478	3739·3 3752·8 3766·4 3780·0 3793·7 3807·3 3821·0 3834·7	216.770 217.163 217.555 217.948 218.341 218.733 219.126 219.519	75 18 1438 1258 31478	4417.9 4432.6 4447.4 4462.2 4477.0 4491.8 4506.7 4521.5	235.619 236.012 236.405 236.798 237.190 237.583 237.976 238.368	SI 18 14 38 12 12 15 8 34 7 7 8	5153.0 5168.9 5184.9 5200.8 5216.8 5232.8 5248.9 5264.9	254.469 254.862 255.254 255.647 256.040 256.433 256.825 257.218

Areas and Circumferences of Circles from 1 to 100

	AREAS A	IND CIRCU	MFE	RENCES	OF CIRCLE	22 FK	OM I TO	100
Dia.	Area	Circum.	Dia.	Area	Circum.	Dia.	Area	Circum.
্থ প্ৰত্যাধনতাত নথেচাত জাখানত প্ৰ	5281.0 5297.1 5313.3 5329.4 5345.6 5361.8 5378.1 5394.3	257.611 258.003 258.396 258.789 259.181 259.574 259.967 260.359	88 नळ नंबाज्यक नाताक्रीक्षण्यान्क	6082.1 6099.4 6256.7 6134.1 6151.4 6168.8 6186.2 6203.7	276.460 276.853 277.846 277.638 278.031 278.424 278.816 279.209	94 1814381215833478	6939.8 6958.2 6976.7 6995.3 7013.8 7032.4 7051.0 7069.6	295.310 295.702 296.095 296.488 296.881 297.273 297.666 298.059
প্র ন্ধার্তি নিয়াত্রতাধান্ত প্র	5410.6 5426.9 5443.3 5459.6 5476.0 5492.4 5508.8 5525.3	260.752 261.145 261.538 261.930 262.323 262.716 263.103 263.501	89 18 14 38 12 58 3 4 7 8	6221.I 6238.6 6256.I 6273.7 6291.2 6308.8 6326.4 6344.I	279.602 279.994 280.387 280.780 281.173 281.565 281.958 282.351	95 181438 12158 314718	7088.2 7106.9 7125.6 7144.3 7163.0 7181.8 7200.6 7219.4	298.451 298.844 299.237 299.629 300.022 300.415 300.807 301.200
84 181430012000000000000000000000000000000000	5541.8 5558.3 5574.8 5591.4 5607.9 5624.5 5641.2 5657.8	263.80,4 264.286 264.679 265.072 265.465 265.857 266.250 266.643	90 1/8 1/4 3/8 1/215/8 3/417/8	6361.7 6379.4 6397.1 6414.9 6432.6 6450.4 6468.2 6486.0	282.743 283.136 283.529 283.921 284.314 284.707 285.100 285.492	96 181438121583478	7238.2 7257.1 7276.0 7294.9 7313.8 7332.8 7351.8 7370.8	301.593 301.986 302.378 302.771 303.164 303.556 303.949 304.342
85 18 14 19 16 16 16 16 16 16 16 16 16 16 16 16 16	5674.5 5691.2 5707.9 5724.7 5741.5 5758.3 5775.1 5791.9	267.035 267.428 267.821 268.213 268.606 268.999 269.392 269.784	9 I 1/8 1/4/3/8 1/2/5/8 3/4/7/8	6503.9 6521.8 6539.7 6557.6 6575.5 6593.5 6611.5 6629.6	285.885 286.278 286.670 287.063 287.456 287.848 288.241 288.634	97 18 14 38 12 58 34 77 8	7389.8 7408.9 7428.0 7447.1 7466.2 7485.3 7504.5 7523.7	304.734 305.127 305.520 305.913 306.305 306.698 307.091 307.483
86	5808.8 5825.7 5842.6 5859.6 5876.5 5893.5 5910.6 5927.6	270.177 270.570 270.962 271.355 271.748 272.140 272.533 272.926	92 18 14 38 12 58 34 78	6647.6 6665.7 6683.8 6701.9 6720.1 6738.2 6756.4 6774.7	289.027 289.419 289.812 290.205 290.597 290.990 291.383 291.775	98 18 14 38 125 8 34 7 8	7543.0 7562.2 7581.5 7600.8 7620.1 7639.5 7658.9 7678.3	307.876 308.269 308.661 309.064 309.447 309.840 310.232 310.625
87 HIGH CHONON CONTROL OF THE CONTRO	5944.7 5961.8 5978.9 5996.0 6013.2 6030.4 6047.6 6064.9	273.319 273.711 274.104 274.497 274.889 275.282 275.675 276.067	93 18 14 33 8 12 5 8 3 4 7 8	6792.9 6811.2 6829.5 6847.8 6866.1 6884.5 6902.9	292.168 292.561 292.954 293.346 293.739 294.132 294.524 294.917	99 18 14 38 125 8 34 7 8	7697.7 7717.1 7736.6 7756.1 7775.6 7795.2 7814.8 7834.4	311.018 311.410 311.803 312.196 312.588 312.981 313.374 313.767

Areas and Circumferences of Circles from 100 to 1000

Diam.	Area.	Circum.	Diam.	Area	Circum.	Diam.	Area	Circum.
100	7853.98	314.16	150	17671.46	471.24	200	31415.93	628.32
101	8011.85	317.30	151	17907.86	474.38	201	31730.87	631.46
102	8171.28	320.44	152	18145.84	477.52	202	32047.39	634.60
103	8332.20	323.58	153	18385.39	480.66	203	32365.47	637.74
104	8494.87	326.73	154	18626.50	483.81	201	32685.13	64 0 .88
105	8650.01	329.87	155	18869.19	486.95	205	33006.36	644.03
106	8824.73	333.01	156	19113.45	490.09	206	33329.16	647.17
107	8992.02	336.15	157	19359.28	493.23	207	33653.53	650.31
108	9160.88		158	19606.68	496.37 .	208	33979-47	653-45
109	9331.32		159	19855.65	499.51	209	34306.98	656 59
110	9503.32	345.58	160	20106.10	502.65	210	34636.06	659.73
111	9503.32		161	20358.31	505.80	211	34966.71	662.88
111	9852.03		162	20611.00		212	35298.94	
	10028.75		163	20867.24		213	35632.73	
113	100207.03		164	21124.07	515.22	214	35968.00	
114				21124.07		215	36305.03	
115	10386.89		165					
116	10568.32		166	21642.43		216	36643.54	
117	10751.32		167	21903.97		217	36983.61	
118	10935.88		168	22167.08		218	37325.26	
119	11122.02	373.85	169	22.131.76	530.93	219	37668.48	688.or
120	11300.73	376.99	170	22698.01	534.07	220	38013.27	691.15
121	11,100.01	380.13	171	22965.83	537.21	221	38359.63	
I 2 2	11689.87	383.27	172	23235.22	540.35	222	38707.56	697.43
123	11882.20		173	23506.18	543.50	223	39057.07	700.58
124	12076.28	380.56	174	23778.71	546.64	224	30408.14	703.72
125	12271.85		175	24052.82	549.78	225	39760.78	706.86
126	12468.08		176	24328.49	552.92	226	40115.00	710.00
127	12667.60		177	24605.74	556.06	227	40470.78	713.14
128	12867.06		178	24884.56		228	40828:14	
129	13069.81		179	25164.94		229	41187.07	
130	13273.23	408.41	180	25446.90	565.49	230	41547.56	722.57
131	13478.22		181	25730.43		231	41000.63	
	13684.78	411.53	182	26015.53		232	42273.27	
132	13802.01		183	26302.20		233	42638.48	
133			184	26500.44		234	43005.26	
134	14102.61		185	26880.25	581.19		43373.61	
135	14313.88		186			235		
136	14526.72			27171.63			43743.54	
137	14741.14		187	27464.59		237	44115.03	
138	14957.12		188	27759.11		238	44488.00	
139	15174.68	436.68	189	28055.21	593.76	239	44862.73	750.84
140	15393.80	439.82	190	28352.87	596.90	240	45238.93	
141	15614.50	442.96	191	28652.11		24I	45616.71	
142	15836.77		192	28952.92	603.19	242	45996.06	
143	16060.61		193	29255.30		243	46376.98	763.41
144	16286.02		194	29559.25		244	46759.47	
145	16513.00		195	29864.77		245	47143.52	
146	16741.55		196	30171.86		246	47529.16	
147	16971.67		197	30480.52		247	47916.36	
148	17203.36		108	30790.75		248	48305.13	
140	17436.62		199	31102.55		240	48695.47	
149	1/430.02	1 400.10	199	3.102.33	023.20	-49	75593147	'

AREAS AND CIRCUMFERENCES OF CIRCLES FROM 100 TO 1000

Diam.	Area	Circum.	Diam.	Area	Circum.	Diam.	Area	Circum.
250	49087.39	785.40	300	70685.83	942.48	350	96211.28	
251	49480.87	788.54	301	71157.86	945.62	351	96761.84	1102.70
252	49875.92	791.68	302	71631.45	948.76	352	97313.97	1105.84
253	50272.55	794.82	303	72106.62	951.90	353	97867.68	1108.98
254	50670.75	797.96	30.4	72583.36	955.04	354	98422.96 98979.80	
255	51070.52 51471.85	801.11	305 306	73061.66 73541.54	958.19 961.33	355 356	99538.22	
250 257	51874.76	807.39	307	73541.54	964.47	357	100008.21	1121.55
258	52279.24	810.53	308	74522.99	967.61	358	100650.77	1124.60
259	52685.29	813.67	309	74990.60	970.75	359	101222.90	1127.83
260	53092.92	816,81	310	75476.76	973.89	360	101787.60	1130.97
261	53502.11	819.96	311	75964.50	977.04 080.18	361	102353.87	1134.11
262 263	53912.87	823.10 826.24	312	76453.80	983.32	362 363	102921.72	
264	54325.21 54739.11	820.24	313	76944.67 77437.12	086.46	364	103491.13	
265	55154.59	832.52	315	77931.13	989.60	365	104634.67	1146.68
266	55571.63	835.66	316	78426.72	992.74	366	105208.80	
267	55990.25	838.81	317	78923.88	995.88	367	105784.49	1152.96
268	56410.44	841.95	318	79422.60	999.03	368	106361.76	
269	56832.20	845.09	319	79922.90	1002.17	369	106940.60	1159.25
270	57255.53	848.23	320	80424.77	1005.31	370	107521.01	
271	57680.43	851.37	321	80928.21	1008.45	371	108102.99	1165.53
272	58106.90 58534.94		322	81433.22 81939.80	1011.59	372 373	100030.34	1171.81
273 274	58964.55	860.80	,323 324	82447.96	1017.88	374	100858.35	1174.96
275	59395.74		325	82957.68	1021.02	375	110446.62	
276	50828.40	867.08	326	83468.98	1024.16	376	111036.45	1181.24
277	60262.82	870.22	327	83981.84	1027.30	377	111627.86	1184.38
278	60698.71		328	84496.28		378	112220.83	1187.52
279	61136.18	876.50	329	85012.28	1033.58	379	112815.38	1190.66
280 281	61575.22		330	85529.86	1036.73	380 381	113411.49	1193.81 1196.95
282	62015.82		331	86569.73	1039.87	382	114608.44	
283	62901.75		332 333	87002.02	1045.01	383	115200.27	1203.23
284	63347.07	802.21	334	87615.88	1040.20	384	115811.67	1206.37
285	63793.97	895.35	335	88141.31	1052.43	385	116415.64	1209.51
286	64242.43	898.50	336	88668.31	1055.58	386	117021.18	
287	64692.46	901.64	337	89196.88	1058.72	387	117628.30	
288	65144.07	904.78	338	89727.03	1061.86	388	118236.98	
289	65597.24	907.92	339	90258.74	1065.00	389	118847.24	1222.08
290	66551.99		340	90792.03	1068.14	390	119459.06	
291	66508.30		341	91326.88	1071.28	391 392	120072.46	
292 293	67425.65		342 343	02401.31	1074.42	392	121303.96	
293	67886.68		343	92401.31	1080.71	393	121022.07	
295	68349.28		345	93482.02	1083.85	395	122541.75	
296	68813.45		346	94024.73	1086.99	396	123163.00	1244.07
297	69279.19		347	94569.01	1000.13	397	123785.82	
298	69746.50	936.19	348	95114.86		398	124410.21	
299	70215.38	939-34	349	95662.28	1096.42	399	125036.17	1253.50

Areas and Circumferences of Circles from 100 to 1000

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Diam.	Area	Circum.	Diam.	Area	Circum.	Diam.	Area	Circm.
								0-
400	125663.71	1256.64	450	159043.13	1413.72	500	196349.54	1570.80
401	126292.81	1259.78	451	159750.77	1416.86	501	197135.72	1573.94
402	126923.48	1262.92	452	160459.99	1420.00	502	197923.48	1577.08
403	127555.73	1266.06	453	161170.77	1423.14	503	198712.80	
404	128189.55	1269.20	454	161883.13	1426.28	504	199503.70	
405	128824.93	1272.35	455	162597.05	1429.42	505	200296.17	
406	129461.89	1275.49	456	163312.55	1432.57	506	201000.20	
407	130100.42		457	164029.62	1435.71	-507	201885.81	
408	130740.52	1281.77	458	164748.26	1438.85	508	202682.99	
409	131382.19	1284.91	459	165468.47	1441.99	509	203481.74	1599.07
410	132025.43	1288.05	460	166190.25	1445.13	510	204282.06	
411	132670.24	1291.19	461	166913.60		511	205083.95	
412	133316.63	1294.34	462	167638.53	1451.42	512	205887.42	
413	133964.58	1297.48	463	168365.02	1454.56	513	206692.45	
414	134614.10	1300.62	464	169093.08	1457.70	514	207499.05	
415	135265.20	1303.76	465	169822.72	1460.84	515	208307.23	
416	135917.86	1306.90	466	170553.92	1463.98	516	209116.97	1621.06
417	136572.10	1310.04	467	171286.70	1467.12	517	209928.20	1624.20
418	I37227.QI	1313.10	468	172021.05	1470.27	518	210741.18	1627.34
419	137885.29	1316.33	469	172756.97	1473.41	519	211555.63	1630.49
420	138544.24	1319.47	470	173494.45	1476.55	520	212371.66	1633.63
421	130204.76	1322.61	471	174233.51	1479.69	521	213189.26	1636.77
422	130866.85	1325.75	472	174974.14		522	214008.43	1639.91
423	140530.51	1328.89	473	175716.35		523	214829.17	1643.05
424	141195.74		474	176460.12	1480.11	524	215651.40	1646.19
425	141862.54		475	177205.46		525	216475.37	1649.34
426	142530.92	1338.32	476	177952.37	1495.40	526	217300.82	
427	143200.86		477	178700.86		527	218127.85	
428	143872.38		478	179450.91		528	218056.44	
429	144545.46		479	180202.54		529	219786.61	
430	145220.12	1350.88	480	180955.74	1507.96	530	220618.34	1665.04
431	145896.35		48r	181710.50		531	221451.65	1668.10
432	146574.15		482	182466.84		532	222286.53	
433	147253.52		483	183224.75	1517.39	533	223122.08	1674.47
434	147934.46		484	183984.23	1520.53	534	223061.00	1677.61
435	148616.07		485	184745.28		535	224800.59	1680.75
436	140301.05		486	185507.90		536	225641.75	1883.89
437	149986.70		487	186272.10		537	226484.48	
438	150673.93		488	187037.86		538	227328.79	
439	151362.72		489	187805.19		539	228174.66	
440	152053.08	1382.30	490	188574.10	1539.38	540	220022.10	1696.46
441	152745.02		491	189344.57		541	220871.12	
442	153438.53	1388.58	492	100116.62		542	230721.71	
443	154133.60		493	190890.24		543	231573.86	
443	154830.25		493	101665.43	1551.95	544	232427.59	
444	155528.47		494	192442.18		545	233282.80	
445	156228.26		495	193220.51		546	234139.76	
	156929.62		497	193220.31		547	234998.20	
447 448	157632.55		497	194781.89		548	235858.21	
	158337.06		499	195564.93		549	236719.79	
449	130337.00	1410.50	499	193304.93	2307.03	349	230/19.79	-,-4.73

Areas and Circumferences of Circles from 100 to 1000

237582.94 238447.67 239313.96 240181.83 241051.26 241922.27 242704.85 243668.99 244544.71 245422.00 246300.86 247181.30	1727.88 1731.02 1734.16 1737.30 1740.44 1743.58 1746.73 1749.87 1753.01 1756.15	600 601 602 603 604 605 606 607 608 609	282743.34 283686.60 284631.44 285577.84 286525.82 287475.36 288426.48 289379.17 290333.43 291289.26	1884.96 1888.10 1891.24 1894.38 1897.52 1900.66 1903.81 1906.95	650 651 652 653 654 655 656	331830.72 332852.53 333875.90 334900.85 335927.36 336955.45 337985.10	2042.04 2045.18 2048.32 2051.46 2054.60 2057.74
239313.96 240181.83 241051.26 241922.27 242794.85 243668.99 244544.71 245422.00 246300.86 247181.30	1734.16 1737.30 1740.44 1743.58 1746.73 1749.87 1753.01 1756.15	602 603 604 605 606 607 608	284631.44 285577.84 286525.82 287475.36 288426.48 289379.17 290333.43	1891.24 1894.38 1897.52 1900.66 1903.81	652 653 654 655 656	333875.90 334900.85 335927.36 336955.45	2048.32 2051.46 2054.60
240181.83 241051.26 241922.27 242704.85 243668.99 244544.71 245422.00 246300.86 247181.30	1737.30 1740.44 1743.58 1746.73 1749.87 1753.01 1756.15	603 604 605 606 607 608	285577.84 286525.82 287475.36 288426.48 289379.17 299333.43	1894.38 1897.52 1900.66 1903.81	653 654 655 656	334900.85 335927.36 336955.45	2051.46 2054.60
241051.26 241922.27 242794.85 243668.99 244544.71 245422.00 246300.86 247181.30	1740.44 1743.58 1746.73 1749.87 1753.01 1756.15	604 605 606 607 608	286525.82 287475.36 288426.48 289379.17 299333.43	1897.52 1900.66 1903.81	654 655 656	335927.36 336955.45	2054.60
241922.27 242794.85 243668.99 244544.71 245422.00 246300.86 247181.30	1743.58 1746.73 1749.87 1753.01 1756.15	605 606 607 608	287475.36 288426.48 289379.17 299333.43	1900.66	655 656	336955.45	
242794.85 243668.99 244544.71 245422.00 246300.86 247181.30	1746.73 1749.87 1753.01 1756.15	606 607 608	288426.48 289379.17 290333.43	1903.81	656		2057.71
243668.99 244544.71 245422.00 246300.86 247181.30	1749.87 1753.01 1756.15	607 608	289379.17 290333.43		050	337005.10	
244544.71 245422.00 246300.86 247181.30	1753.01 1756.15	608	290333.43	1900.95		339016.33	2060.88
245422.00 246300.86 247181.30	1756.15			1010.00	657 658	340049.13	2064.03
247181.30	1750 20		291209.20	1913.23	659	341083.50	2070.31
247181.30	-139.29	610	292246.66	1916.37	660	342119.44	2073.45
		611	293205.63		661	343156.95	2076.59
							2079.73 2082.88
							2086.02
							2080.02
251607.01	1778.14	616			666		2002.30
252496.87	1781.28	617			667		2005.44
253388.30	1784.42	618	299962.41	1941.50	668	350463.51	2098.58
254281.29	1787.57	619	300933.95	1944.65	669	351513.59	2101.73
255175.86	1790.71	620	301907.05	1947.79	670	352565.24	2104.87
							2111.15
258760.85	1803.27						2117.43
259672.27	1806.42	625	306796.16	1963.50	675	357847.04	2120.58
	1809.56	626	307778.69	1966.64	676	358908.11	2123.72
					677		2126.86
263297.67	1815.84	628	309748.47 310735.71	1972.92	679	361034.97 362100.75	2130.00
264207.01	1822.12	630	311721.53	1070.20	680	363168.11	2136.28
					68r		2139.42
266033.21	1828.41	632	313706.88	1985.49	682	365307.54	2142.57
		633	314700.40	1988.63		366379.60	2145.71
							2148.85
		035					2151.99
							2155.13 2158.27
							2150.27
272471.12	1850.40	639	320694.56	2007.48	689	372845.00	2164.56
273397.10	1853.54	640	321699.09	2010.62	690	373028.07	2167.70
			322705.18	2013.76	691	375012.70	2170.84
							2173.98
							2177.12
							2180.27 2183.41
							2186.55
							2180.60
280861.52	1878.67	648	329791.83	2035.75	698	382649.13	2192.83
281801.65	1881.81	649	330810.49	2038.89	699	383746.33	2195.97
	24806.3,3 2489.6.87 2498.3.01 259718.73 2498.3.01 252496.87 255175.86 25027.20 255175.86 25027.27 264207.22 2651481.83 262388.96 263297.67 264207.94 265110.79 26623.21 26693.21 273397.10 27471.12 273397.10 274321.67 274321.66 27523.78 271834.89 272783.78 272471.12	248063.30 248063.30 248063.30 248046.87 249832.01 251607.01 1775.00 251607.01 1775.10 2524281.29 1787.57 255175.86 2524281.29 1787.57 255175.86 1790.71 2509672.20 257868.99 1800.13 258760.85 2621481.83 1812.70 262388.96 1815.84 263297.67 1818.08 264207.04 265110.79 262388.96 26446.76 26447.60 26540.30 26427.04 265110.79 26238.96 26427.04 265110.79 26238.96 27127 27337.10 27337.1	248063.30 1765.58 612 248946.87 1768.72 613 249832.01 1771.86 614 250718.73 1775.00 614 252748.73 1775.00 615 251607.01 1778.74 616 252406.87 1781.28 617 252496.87 1781.28 618 253388.30 1784.42 618 2554281.29 1787.57 619 255175.86 1700.71 622 255075.86 1700.71 622 257868.99 1800.13 623 258760.85 1803.27 624 250576.26 1800.56 626 261481.83 1812.70 627 262388.06 1815.84 626 262407.04 1822.12 630 26329.76 1818.98 629 264207.04 1822.12 630 266033.21 1828.41 632 266033.21 1828.41 632 266782.89 1837.85 633 26702.59 1840.07 636 27043.86 1844.11 637 27043.86 1844.11 637 27154.670 1847.26 638 272471.12 1850.40 638 272471.12 1850.80 640 273397.10 1853.54 274324.66 1856.68 641 27116.75 1866.11 641 278050.58 1860.25 645 278050.58 1860.25 645 278050.58 1860.25 645 278050.58 1860.25 645 278050.58 1860.25 645 278050.58 1860.25 645 278050.58 1860.25 645 278050.58 1860.25 645 278050.58 1860.25 645 278050.58 1860.25 645	243603.30 1765.58 612 294166.17 24894.68.17 1768.72 613 205128.28 24983.2.01 1771.86 614 206001.97 250718.73 1775.20 615 207507.25 22406.87 1778.14 616 208024.05 252406.87 1778.128 617 208092.44 253388.30 1784.42 618 209062.41 254281.29 1787.57 619 300933.05 255175.86 1700.71 620 301007.05 620 200.20 1703.85 621 302881.73 25609.21 1706.90 622 303857.08 257680.89 1800.13 623 304835.80 258609.21 1706.90 622 303857.08 257608.51 1803.27 624 305815.20 25072.27 1806.12 625 306706.16 626 307778.69 626 30778.69 626 30778.69 627 303578.17 620 627 303578.17 620 627 303579.18 628 628 628 628 628 628 628 628 628 62	243663.30 1765.58 612 294166.17 1922.65 24894.68.7 1768.72 613 295128.28 1925.80 24983.01 1771.86 614 297627.22 1932.30 251607.01 1778.14 616 297627.22 1932.30 252496.87 1781.28 617 298022.45 1938.36 297627.21 1932.30 253388.30 1781.42 618 299622.41 1938.36 20 255175.86 1790.71 619 302881.73 195-09 25606.27.1 1790.90 622 33857.98 195-40 257688.99 1800.13 623 303835.98 1957-21 25607.2.27 1806.12 625 30778.60 1960.35 262376.96 180.9.6 626 30778.60 1960.35 260576.26 180.9.6 626 30778.60 1960.35 264207.04 1822.12 630 31721.453 1970.00 265110.79 1828.41 632 313706.88<	24396.3-30 1765.5-8 612 294166.17 1922.6-5 662 24894.8-7 1768.7-2 613 29728.7-2 1932.3-6 663 24983.0-1 1771.86 614 296091.97 1928.0-1 664 297087.7-2 1932.0-5 665 297087.7-2 1932.0-5 665 297087.7-2 1932.0-5 665 297087.7-2 1932.0-5 665 297087.7-2 1932.0-5 665 297087.7-2 1932.0-5 665 297087.7-2 1932.0-5 665 297087.7-2 1932.0-5 665 297087.7-2 1932.0-5 665 297087.7-2 1932.0-5 665 297087.7-2 1932.0-5 665 297087.7-2 1932.0-5 665 297087.7-2 1932.0-5 665 297082.1-1 1938.3-5 667 297092.1-1 1938.3-5 667 297092.1-1 1941.5-5 668 300933.9-5 1944.6-5 669 255175.86 1790.71 620 301907.0-5 1947.7-9 670 2557868.99 1800.13 623 303857.98 1954.0-7 672 257868.99 1800.13 623 303857.98 1954.0-7 672 257868.99 1800.13 623 303857.98 1954.0-7 672 257868.99 1800.13 623 30778.6-6 1960.3-5 674 2506576.2-6 1820.5-6 626 307778.6-6 1960.3-5 674 26238.8-6 1815.8-4 628 30778.4-7 1972.0-2 678 262327.6-7 1884.9-8 629 30778.5-6 1966.6-4 676 2624207.0-4 1822.1-2 630 311724.5-3 1970.2-0 680 26329.7-6 1882.8-4 628 30778.5-6 1966.6-4 676 679 26369.3-2 1885.8-4 628 30778.5-6 1968.5-4 682 205110.7-9 1825.2-7 631 313706.88 1085.4-9 682 205110.7-9 1825.2-7 631 313706.88 1085.4-9 682 27714.1-2 1850.4-9 636 31470.0-4 1988.6-3 683 27768.5-6 1841.1-6 37 38609.2-3 1994.0-7 684 27714.1-2 1850.4-9 636 31272.0-9 1994.9-7 684 27714.1-7 1866.1-1 637 38609.2-3 1994.9-7 684 27714.1-7 1866.1-1 637 38609.2-3 1094.9-7 684 27714.1-7 1866.1-1 637 38609.2-3 1094.9-7 684 27714.1-7 1866.1-1 637 38609.2-3 1094.9-7 684 27714.1-7 1866.1-1 637 38609.2-3 1094.9-7 684 27714.1-7 1866.1-1 637 38609.2-3 1094.9-7 684 27714.5-1 1850.4-5 645 32771.8-5 2010.0-0 692 274324.6-6 1850.6-6 643 32770.9-0 200.0-6 603 32770.9-2 1875.5-3 647 32774.7-4 2032.6-1 697 278050.5-8 1875.5-3 647 32774.7-4 2032.6-1 697 278050.5-8 1875.5-3 647 32774.7-4 2032.6-1 697 278050.5-8 1875.5-3 647 32774.7-4 2032.6-1 697 27902.9-7 1875.5-3 647 32774.7-4 2032.6-1 697 27902.9-7 1875.5-3 647 32774.7-4 2032.6-1 697 27902.9-7 1875.5-3 647 32774.7-4 2032.6-1 697 27902.9-7 1875.5-3 647 32774.7-4 2032.6-1 697 27902.9-7 1875.5	243863.30 1765.58 612 294166.17 1922.65 662 344166.23 24894.68.7 1768.72 613 295182.88 1925.80 663 345236.69 249832.01 1771.86 614 296091.97 1928.94 664 346278.91 250718.73 1775.00 615 297057.22 1932.08 665 34368.08.25 25169.70 1778.14 616 298024.05 1935.22 666 348368.07 252496.87 1778.12 617 298902.44 1938.36 667 349415.00 253388.30 1784.42 618 299062.41 1941.50 668 350463.55 254281.29 1787.57 619 300933.05 1944.05 669 351513.59 255175.86 1790.71 620 301907.05 1947.79 670 352565.24 326969.71 1796.90 622 303857.08 1951.07 672 353618.32 254288.29 1780.35 621 302881.73 1950.93 671 353618.45 25768.99 1800.13 623 304835.80 1957.21 673 355473.24 25768.99 1800.12 625 305765.16 1963.50 675 3357847.04 256576.26 1800.56 626 307778.69 1966.04 676 38908.11 622 303778.69 1966.04 676 38908.11 622 303778.69 1966.04 676 38908.11 622 303778.09 1960.60 679 362100.07 622 303875.79 1969.78 677 339070.75 262388.96 1815.84 628 309748.47 1972.92 678 361034.07 266033.21 1828.41 628 309748.47 1972.92 678 361034.07 266038.21 1828.41 632 311724.53 1970.20 680 363168.11 266038.21 1828.41 632 313706.88 1985.49 682 365307.60 1985.49 682 365307.60 1847.04 1831.55 633 314700.40 1988.63 683 3604237.04 267864.76 1834.69 634 31509.25 1991.77 684 307435.24 26783.21 1850.40 639 31274.04 1988.03 683 3603370.60 2774324.66 1834.69 634 31509.25 1991.77 684 307435.24 27052.59 1840.97 636 317600.42 1908.05 686 37283.50 2774324.66 1834.69 634 31509.25 1991.77 684 307435.24 27052.59 1840.97 636 317600.42 1908.05 686 37283.50 2774324.66 1836.04 644 32573.28 2001.19 685 36528.84 27714.07 1850.40 634 32779.02 201.00 690 37302.50 2774324.66 1836.01 644 32573.28 2020.44 683 37760.32 27805.58 1850.82 642 323772.59 2020.44 683 377286.03 377856.03 378276.03 378297.99 1840.97 648 32772.20 200.44 693 377360.31 377360.31 372760.33 378276.03 377856.14 327902.29 1840.97 643 327790.22 200.44 693 377360.33 3722.00 643 32779.22 2020.44 693 377360.33 3722.00 643 327922.97 1855.53 647 327790.22 2020.44 693 377360.33 3772.20 2020.49 63 377360.33 3772.00 37780.97 648 327790

Areas and Circumferences of Circles from 100 to 1000

Diam.	Area	Circum.	Diam.	Area	Circum.	Diam.	Area	Circum.
700	384845.10		7.50	441786.47	2356.10	800	502654.82	2552.05
		2199.11	750	441760.47		801	503012.25	2513.27
701	385945.44	2202.26	75I		2359.34			2516.42
702	387047.36	2205.40	752	444145.80	2362.48	802	505171.24	2519.56
703	388150.84	2208.54	753	445327.83	2365.62	803	506431.80	2522.70
704	389255.90	2211.68	754	446511.42	2368.76	804	507693.94	2525.84
705	390362.52	2214.82	755	447696.59	2371.90	805	508957.64	2528.98
706	391470.72	2217.96	756	448883.32	2375 04	806	510222.92	2532.12
707	392580.49		757	450071.63	2378.10	807	511489.77	2535.27
708	393691.82	2224.25	758	451261.51	2381.33	808	512758.19	2538.41
709	394804.73	2227.39	759	452452.96	2384.47	809	514028.18	2541.55
710	395919.21	2230.53	760	453645.98	2387.61	810	515299.74	2544.69
711	397035.26	2233.67	761	454840.57	2390.75	811	516572.87	2547.83
712	398152.89	2236.81	762	456036.73	2393.89	812	517847.57	2550.97
713	399272.08	2239.96	763	457234.46	2397.04	813	519123.84	2554.11
714	400392.84	2243.10	764	458433.77	2400.18	814	520401.68	2557.26
715	401515.18	2246.24	765	459634.64	2403.32	815	521681.10	2560.40
716	402639.08	2249.38	766	460837.08	2406.46	816	522962.08	2563.54
717	403764.56	2252.52	767	462041.10	2409.60	817	524244.63	2566.68
718	404891.60	2255.66	768	463246.60	2412.74	818	525528.76	2569.82
719	406020.22	2258.81	769	464453.84	2415.88	819	526814.46	2572.96
720	407150.41	2261.05	770	465662.57	2410.03	820	528101.73	2576.11
721	408282.17	2265.00	771	466872.87	2422.17	821	529390.56	2579.25
722	400415.50		772	468084.74	2425.31	822	530680.07	2582,30
723	410550.40		773	469298.18	2428.45	823	531972.95	2585.53
724	411686.87	2274.51	774	470513.10	2431.50	824	533266.50	2588.67
725	412824.01	2277.65	775	471729.77	2434.73	825	534561.62	2591.81
726	413964.52	2280.80	776	472947.92	2437.88	826	535858.32	2594.96
727	415105.71	2283.04	777	474167.65	2441.02	827	537156.58	2508.10
728	416248.46	2287.08	778	475388.94	2444.16	828	538456.41	2601.24
729	417392.79	2290.22	779	476611.81	2447.30	829	539757.82	2604.38
730	418538.68	2203.36	780	477836.24	2450.44	830	541060.70	2607.52
731	410686.15	2206.50	781	479062.25	2453.58	83I	542365.34	2610.66
732	420835.10	2200.65	782	480280.83	2456.73	832	543671.46	2613.81
733	421985.79		783	481518.97	2459.87	833	544979-15	2616.05
734	423137.97	2305.03	784	482749.60	2463.01	834	546288.40	2620.00
735	424291.72	2300.07	785	483081.08	2466.15	835	547599-23	2623.23
736	425447.04		786	485215.84	2460.20	836	548911.63	2626.37.
737	426603.94	2315.35	787	486451.28	2472.43	837	550225.61	2620.51
738	427762.40		788	487688.28	2475.58	838	551541.15	2632.65
739	428022.43	2321.64	780	488026.85	2478.72	839	552858.26	2635.80
740	430084.03	2324.78	790	490166.99	2481.86	840 841	554176.94	2638.94 2642.08
741	431247.21	2327.92	791	491408.71	2488.14	842	555497.20 556819.02	
7.42	432411.95	2331.06	792	492651.99				
743	433578.27	2334.20	793	493896.85	2491.28	843	558142.42	2651.50
744	434746.16	2337.34	794	495143.28	2494.42	844	559467.39	
745	435915.62	2340.49	795	496391.27	2497.57	845	560793.92	2654.65
746	437086.64		796	497640.84	2500.71	846	562122.03	2657.79
747	438259.24		797	498891.98	2503.85	847	563451.71	2660.03
748	439433.41		798	500144.69	2506.99	848	564782.96	
749	440609.16	2353.05	799	501398.97	2510.13	849	566115.78	2007.21
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Areas and Circumferences of Circles from 100 to 1000

Diam.	Area	Circum.	Diam.	Area	Circum.	Diam.	Area	Circum.
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850	567450.17	2670.35	900	636172.51	2827.43	950	708821.84	2984.51
851	568786.14	2673.50	100	637587.01	2830.58	951	710314.88	2987.65
852	570123.67	2676.64	Q02	639003.00	2833.72	952	711800.50	2000.80
853	571462.77	2679.78	903	640420.73	2836.86	953	713305.68	2993.94
854	572803.45	2682.02	904	641839.95	2840.00	954	714803.43	2997.08
855	574145.69	2686.06	905	643260.73	2843.14	955	716302.76	3000.22
856	575489.51	2689.20	906	644683.00	2846.28	956	717803.66	3003.36
857	576834.90	2692.34	907	6,16107.01	2849.42	957	710306.12	3006.50
858	578181.85	2695.49	908	647532.51	2852.57	958	720810.16	3000.55
859	579530.38	2698.63	909	648959.58	2855.71	959	722315.77	3012.79
860	580880.48	2701.77	910	650388.22	2858.85	960	723822.95	3015.93
861	582232.15	2704.91	110	651818.43	2861.99	96 1	725331.70	3019.07
862	583585.39	2708.05	912	653250.21	2865.13	962	7 2 6 8 4 2 . 0 2	3022.21
863	584940.20	2711.19	913	• 654683.56	2868.27	963	728353.91	3025.35
864	586296.59	2714.34	914	656118.48	2871.42	964	729867.37	3028.50
865	587654.54	2717.48	915	657554.98	2874.56	965	731382.40	3031.64
866	589014.07	2720.62	016	658993.04	2877.70	966	732899.01	3034.78
867	590375.16	2723.76	917	660432.68	2880.84	967	734417.18	3037.92
868	501737.83	2726.00	918	661873.88	2883.98	068	735936.93	3041.06
869	593102.06	2730.04	010	663316.66	2887.12	969	737458.24	3044.20
			-			1 1		
870	594467.87	2733.19	920	664761.01	2890.27	970	738981.13	3047.34
871	595835.25	2736.33	921	666206.92	2893.41	971	740505.59	3050.49
872	597204.20	2739.47	922	667654.41	2896.55	972	742031.62	3053.63
873	598574.72	2742.61	923	669103.47	2899.69	973	743559.22	3056.77
874	599946.81	2745.75	924	670554.10	2902.83	974	745088.39	3059.91
875	601320.47	2748.89	925	672006.30	2905.97	975	746619.13	3063.05
876	602695.70	2752.04	926	673460.08	2909.II	976	748151.44	3066.19
877	604072.50	2755.18	927	674915.42	2912.26	977	749685.32	3069.34
878	605450.88	2758.32	928	676372.33	2915.40	978	751220.78	3072.48
879	606830.82	2761.46	929	677830.82	2918.54	979	752757.80	3075.62
880	608212.34	2764.60	930	679290.87	2921.68	980	754296.40	3078.76
881	609595.42	2767.74	931	680752.50	2924.82	981	755836.59	3081.90
882	610980.08	2770.88	932	682215.69	2927.96	982	757378.30	3085.04
883	612366.31	2774.03	933	683680.46	2931.11	983	758921.61	3088.19
884	613754.11	2777.17	934	685146.80	2934.25	984	760466.48	3091.33
885	615143.48	2780.31	935	686614.71	2937.39	985	762012.93	3094.47
886	616534.42	2783.45	936	688084.19	2940.53	986	763560.95	3097.61
887	617926.93	2786.59	937	689555.24	2943.67	987	765110.54	3100.75
888	619321.01	2789.73	938	691027.86	2946.81	988	766661.70	3103.89
889	620716.66	2792.88	939	692502.05	2949.96	989	768214.44	3107.04
890	622113.89	2796.02	940	693977.82	2953.10	990	769768.74	3110.18
891	623512.68	2799.16	941	695455.15	2956.24	991	771324.61	3113.32
892	624913.04	2802.30	942	696934.06	2959.38	992	772882.06	3116.46
893	626314.98	2805.44	943	698414.53	2962.52	993	774441.07	3119.60
894	627718.49	2808.58	944	699896.58	2965.66	994	776001.66	3122.74
895	629123.56	2811.73	945	701380.19	2968.81	995	777563.82	3125.88
896	630530.21	2814.87	946	702865.38	2971.95	996	779127.54	3129.03
897	631938.43	2818.01	947	704352.14	2975.09	997	780692.84	3132.17
898	633348.22	2821.15	948	705840.47	2978.23	998	782259.71	3135.31
899	634759.58	2824.29	949	707330.37	2981.37	999	783828.15	3138.45
			1 - 1			1000	785398.16	3141.59
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CIRCUMFERENCES AND DIAMETERS OF CIRCLES

Cir- cum.	Diameter	Cir- cum.	Diameter	Cir- cum.	Diameter	Cir- cum.	Diameter
1	.3183	51	16.2338	101	32.1493	151	48.0648
2	.6366	52	16.5521	102	32.4676	152	48.3831
3	.9549	53	16.8704	103	32.7859	153	48.7014
4	1.2732	54	17.1887	104	33.1042	154	49.0197
5	1.5915	55	17.5070	105	33.4225	155	49.3380
6	1.9099	56	17.8254	106	33.7408	156	49.6563
7 8	2.2282	57	18.1437	107	34.0592	157	49.9747
	2.5465	58	18.4620	108	34-3775	158	50.2930
9	2.8648	59	18.7803	109	34.6958	159	50.6113
10	3.1831	60	19.0986	110	35.0141	160	50.9296
11	3.5014	61	19.4169	111	25 2224	161	51.2479
12	3.8197	62	19.7352	1112	35.3324 35.6507	162	51.5662
13	4.1380	63	20.0535	113	35.9690	163	51.8845
14	4.4563	64	20.3718	114	36.2873	164	52.2028
15	4.7746	65	20.6001	115	36.6056	165	52.5211
16	5.0030	66	21.0085	116	36.9239	166	52.8394
17	5.4113	67	21.3268	117	37.2423	167	53.1578
18	5.7296	68	21.6451	118	37.5606	168	53.4761
19	6.6479	60	21.0634	110	37.8789	169	53.7944
20	6.3662	70	22.2817	120	38.1972	170	54.1127
			•				
21	6.6845	71	22.6000	121	38.5155	171	54.4310
22	7.0028	72	22.9183	122	38.8338	172	54.7493
23	7.3211	73	23.2366	123	39.1521	173	55.0676
24	7.6394	74	23.5549	124	39.4704	174	55.3859
25	7.9577	75	23.8732	125	39.7887	175	55.7042
26	8.2761	76	24.1916	126	40.1070	176	56.0225
27	8.5944	77 78	24.5099	127	40.4254	177	56.3408
28	8.9127	78	24.8282	128	40.7437	178	56.6592
29	9.2310	79 80	25.1465 25.4648	129 130	41.0620 41.3803	179	56.9775 57.2958
30	9-5493	30	25.4046	130	41.3003	100	37.2950
31	9.8676	81	25.7831	131	41.6986	181	57.6141
32	10.1850	82	26.1014	132	42.0160	182	57.9324
33	10.5042	83	26.4197	133	42.3352	183	58.2507
34	10.8225	8.1	26.7380	134	42.6535	184	58.5690
35	11.1408	85	27.0563	135	42.9718	185	58.8873
36	11.4592	86	27.3747	136	43.2001	186	59.2056
37	11.7775	87 88.	27.6930	137	43.6085	187	59.5239
38	12.0958		28.0113	138	43.9268	188	59.8423
39	12.4141	89	28.3296	139	44.2451	189	60.1606
40	12.7324	90	28.6479	140	44.5634	190	60.4789
41	13.0507	91	28.9662	141	44.8817	101	60.7972
41	13.3690	92	20.2845	142	45.2000	102	61.1155
43	13.6873	93	20.6028	143	45.5183	193	61.4338
44	14.0056	94	20.0211	144	45.8366	194	61.7521
45	14.3239	95	30.2394	145	46.1540	195	62.0704
46	14.6423	96	30.5577	146	46.4732	196	62.3887
47	14.9606	97	30.8761	147	46.7916	197	62.7070
48	15,2789	98	31.1944	148	47.1009	198	63.0254
49	15,5972	99	31.5127	149	47.4282	199	63.3437
50	15.9155	100	31.8310	150	47.7465	200	63.6620
		1 1		1			

RECIPROCALS OF NUMBERS FROM I TO 1000

No.	Reciprocal	No.	Reciprocal	No.	Reciprocal	No.	Reciprocal
1	1.00000000	51	.01960784	101	.00000000	151	.00662252
2	.50000000	52	.01923077	102	.00980392	152	.00657895
3	-33333333	53	.01886792	103	.00070874	153	.00653595
4	.25000000	54	.01851852	104	.00961538	154	.00640351
4	.20000000	55	.01818182	105	.00952381	155	.00645161
5 6	.16666667	56	.01785714	106	.00932301	156	.00641026
7	.14285714	50	.01754386	107	.00043390	157	.00636943
7 8	.12500000	57 58	.01724138	108	.00934379	158	.00632911
9	.11111111		.01694915	100	.00923920	159	.00628931
10	.I00000000	59 60	.01666667	110	.00909091	160	.00625000
11	.00000000	61	.01639344	111	.00000001	161	.00621118
I 2	.08333333	62	.01612903	112	.00892857	162	.00617284
13	.07692308	63	.01587302	113	.00884956	163	.00613497
14	.07142857	6.4	.01562500	114	.00877193	164	.00609756
15	.06666667	65	.01538461	115	.00869565	165	.00606061
16	.06250000	66	.01515151	116	.00862069	166	.00602410
17	.05882353	67	.01492537	117	.00854701	167	.00598802
18	.05555556	68	.01470588	118	.00847458	168	.00595238
19	.05263158	69	.01449275	119	.00840336	169	.00591716
20	.05000000	70	.01428571	120	.008333333	170	.00588235
21	.04761905	71	.01408451	121	.00826446	171	.00584795
22	.04545455	7.2	.01388889	122	.00819672	172	.00581395
23	.04347826	73	.01369863	123	.00813008	173	.00578035
24	.04166667	74	.01351351	124	.00806452	174	.00574713
25	.04000000	7.5	.013333333	125	.0080000	175	.00571429
26	.03846154	76	.01315789	126	.00793651	176	.00568182
27 28	.03703704	77 78	.01298701	127	.00787402	177	.00564972
	.03571429		.01282051		.00781250	178	.00561798
29	.03448276	79 80	.01265823	129	.00775194	179	.00558659
30	.03333333		.01250000	130	.00769231	180	.00555556
31	.03225806	81	.01234568	131	.00763359	181	.00552486
32	.03125000	82	.01219512	132	.00757576	182	.00549451
33	.03030303	83	.01204819	133	.00751880	183	.00546448
34	.02941176	84	.01190476	134	.00746269	184	.00543478
35	.02857143	85	.01176471	135	.00740741	185	.00540540
36	.02777778	86	.01162791	136	.00735294	186	.00537634
37 38	.02702703	87	.01149425	137	.00729927	187	.00534759
38	.02631579	88	.01136364	138	.00724638	188	.00531914
39	.02564103	89	.01123595	139	.00719424	180	.00529100
40	.02500000	90	.01111111	140	.00714286	190	.00526316
41	.02439024	91	.01098901	141	.00709220	101	.00523560
42	.02380952	92	.01086956	142	.00704225	192	.00520833
43	.02325581	93	.01075269	143	.00699301	193	.00518135
44	.02272727	94	.01063830	144	.00604444	194	
45	.0222222	95	.01052632	145	.00689655	195	.00512820
46	.02173913	96	.01041667	146	.00684931	196	.00510204
	.02127660	97	.01030928	147	.00680272	107	.00507614
47 48	.02083333	98	.01020408	148	.00675676	198	.00505051
49	.02040816	99	10101010.	149	.00671141	199	.00502513
50	.02000000	100	.01000000	150	.00666667	200	.00500000
	<u> </u>	-	'		·		

RECIPROCALS OF NUMBERS FROM 1 TO 1000

No.	Reciprocal	No.	Reciprocal	No.	Reciprocal	No.	Reciprocal
		3	2 (
201	.00497512	25I	.00398406	301	.00332226	351	.00284900
202	.00495049	252	.00396825	302	.00331126	352	.00284091
203	.00492611	253	.00395257	303	.00330033	353	.00283286
204	.00100106	254	.00393701	304	.00328047	354	.00282486
205	.00487805	255	.00392157	305	.00327869	355	.00281600
206	.00485437	256	.00390625	306	.00326797	356	.00280800
207	.00483092	257	.00389105	307	.00325733	357	.00280112
		258	.00387597	308	.00324675	358	.00279330
208	.00480769		.00386100			359	.00278551
200	.00478469	259		309	.00323625	360	
210	.00.176190	260	.00384615	310	.00322581	300	.00277778
211	.00473934	261	.00383142	311	.00321543	361	.00277008
212	.00471698	262	.03381679	312	.00320513	362	.00276243
213	.00469484	263	.00380228	313	.00319489	363	.00275482
214	.00167293	264	.00378788	314	.00318471	364	.00274725
215	.00165116	265	.00377358	315	.00317460	365	.00273973
216	.00162963	266	.00375940	316	.00316456	366	.00273224
217	.00460829	267	.00374532	317	.00315457	367	.00272480
218	.00458716	268	.00373134	318	.00314465	368	.00271739
210	.00456621	260	.00371747	319	.00313480	369	.00271003
		270	.00370370	320	.00312500	370	.00270270
220	.00454545	2/0	.00370370	320	.00312300	370	.002/02/0
221	.00452489	271	.00369004	321	.00311526	371	.00269542
222	.00450450	272	.00367647	322	.00310559	372	.00268817
223	.00448430	273	.00366300	323	.00309597	373	.00268006
224	.00446429	274	.00364063	324	.00308642	374	.00267380
225	.00111111	275	.00363636	325	.00307692	375	.00266667
226	.00442478	276	.00362319	326	.00306748	376	.00265957
227	.00442470	277	.00361011	327	.00305810	377	.00265252
228	.00440329	278	.00359712	328	.00304878	378	.00264550
	.00436681	279	.00358423		.00303951	379	.00263852
229		280		329		380	.00263158
230	. 0 0434783	250	.00357143	330	.00303030	300	.00203138
231	.00432900	28I	.00355872	331	.00302115	381	.00262467
232	.00431034	282	.00354610	332	.00301205	382	.00261780
233	.00429184	283	.00353357	333	.00300300	383	.00261097
234	.00427350	284	.00352113	334	.00299401	384	.00260417
235	.00425532	285	.00350877	335	.00298507	385	.00259740
236	.00423729	286	.00349650	336	.00207619	386	.00259067
237	.00421941	287	.00348432	337	.00296736	387	.00258398
238	.00420168	288	.00347222	338	.00295858	388	.00257732
239	.00418410	280	.00346021	339	.00204085	389	.00257069
240	.00416667	290	.00344828	340	.00294118	390	.00256410
241	.00414938	201	.00343643	247	.00293255	391	.00255754
24I 242	.00414938	29I 292	.00343043	341	.00293233	392	.00255102
				3.12	.00292398	393	.00254453
243	.00411523	293	.00341297	343	.00291345		.00253807
244	.00409836	294	.00340136	344		394	.00253165
245	.00408163	295	.00338983	345	.00289855	395	
246	.00106501	296	.00337838	346	.00289017	396	.00252525
2:47	.00404858	297	.00336700	347	.00288184	397	.00251889
248	.00403226	298	.00335570	348	.00287356	398	.00251256
249	.00401606	299	.00334448	349	.00286533	399	.00250627
250	.00400000	300	.00333333	350	.00285714	400	,00250000
		П		11	1	11	1

RECIPROCALS OF NUMBERS FROM 100 TO 1000

	1						
No.	Reciprocal	No.	Reciprocal	No.	Reciprocal	No.	Reciprocal
407	00040377		00227720	FOX	20122601		00.00.00
401	.00249377	451	.00221729	501	.00199601	551	.00181488
402	.00248756	452	.00221239	502	.00199203	552	.00181159
403	.00248139	453	.00220751	503	.00198807	553	.00180832
404	.00247525	454	.00220264	. 204	.00198413	554	.00180505
405	.00246914	455	.00219780	505	.00198020	555	.00180180
406	.00246305	456	.00219298	506	.00197628	556	.00179856
407	.00245700	457	.00218818	507	.00197239	557	.00179533
408	.00245098	458	.00218341	508	.00196850	558	.00179211
409	.00244499	459	.00217865	509	.00196464	559	.00178891
410	.00243902	460	.00217391	510	.00196078	560	.00178571
411	.00243309	461	.00216920	511	.00195695	561	.00178253
412	.00242718	462	.00216450	512	.00195312	562	.00177936
413	.00242131	463	.00215983	513	.00194932	563	.00177620
414	.00241546	464	.00215517	514	.00194552	564	.00177305
415	.00240964	465	.00215054	515	.00194175	565	.00176991
416	.00240385	466	.00214592	516	.00193798	566	.00176678
417	.00239808	467	.00214133	517	.00193424	567	.00176367
418	.00239234	468	.00213675	518	.00103050	568	.00176056
410	.00238663	469	.00213220	510	.00192678	569	.00175747
420	.00238095	470	.00212766	520	.00192308	570	.00175439
421	.00237530	471	.00212314	521	.00191939	571	.00175131
422	.00236967	472	.00211864	522	.00101571	572	.00174825
423	.00236407	473	.00211416	523	.00191205	573	.00174520
424	.00235849	474	.00210070	524	.00190840	574	.00174216
425	.00235294	475	.00210526	525	.00190476	575	.00173913
426	.00233294	476	.00210081	526	.00190114	576	.00173611
427	.00234192	477	.00210004	527	.00189753	577	.00173310
427	.00233645	478	.002000205	528	.00189753	578	.00173010
429	.00233100	479	.00208768	520	.00180036	579	.00172712
430	.00232558	480	.00208333	530	.00188679	580	.00172414
						-	
43I	.00232019	481	.00207900	531	.00188324	581	.00172117
432	.00231481	482	.00207469	532	.00187970	582	.00171821
433	.00230947	483	.00207039	533	.00187617	583	.00171527
434	.00230415	484	.00206612	534	.00187266	584	.00171233
435	.00229885	485	.00206186	535	.00186916	585	.00170940
436	.00229358	486	.00205761	536	.00186567	586	.00170648
437	.00228833	487	.00205339	537	.00186220	587	.00170358
438	.00228310	488	.00204918	538	.00185874	588	.00170068
439	.00227790	489	.00204499	538	.00185528	589	.00169779
440	.00227273	490	.00204082	540	.00185185	590	.00169491
44 I	.00226757	491	.00203666	541	.00184843	591	.00169205
442	.00226244	492	.00203252	542	.00184502	592	.00168919
443	.00225734	493	.00202840	543	.00184162	593	.00168634
444	.00225225	494	.00202420	544	.00183823	594	,00168350
445	.00224719	495	.00202020	545	.00183486	595	.00168067
446	.00224215	496	.00201613	546	.00183150	596	.00167785
447	.00223714	497	.00201207	547	.00182815	597	.00167504
448	.00223214	498	.00200803	548	.00182482	598	.00167224
449	.00223214	499	.00200303		.00182140		.00167224
450	.0022221	500	.00200401	549 550	.00182149	599 600	.00166667
		<u> </u>					

RECIPROCALS OF NUMBERS FROM I TO 1000

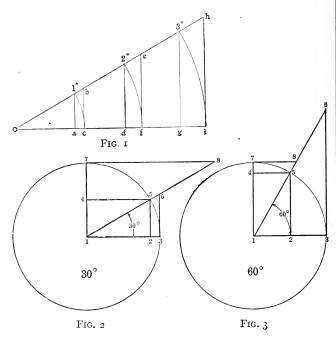
No.	Řeciprocal	No.	Reciprocal	No.	Reciprocal	No.	Reciprocal
601	;00166380	651	.00153610	701	,00142653	751	.00133156
602	.00166113	652	.00153374	702	.00142450	752	.00132979
603	.00165837	653	.00153140	703	.00142247	753	.00132802
604	.00165563	654	.00152905.	704	.00142045	754	.00132626
605	.00165280	655	.00152672	705	.00141844	755	.00132450
606	.00165016	656	.00152439	706	.00141643	756	.00132275
607	.00164745	657	.00152207	707	.00141443	757	.00132100
608	.00164474	658	.00151975	708	.00141243	758	.00131926
600	.00164204	659	.00151745	700	.00141044	759	.00131752
610	.00163934	660	.00151515	710	.00140845	760	.00131579
611	.00163666	661	.00151286	711	.00140647	761	.00131406
612	.00163399	662	.00151057	712	.00140449	762	.00131234
613	.00163132	663	.00150830	713	.00140252	763	.00131062
614	.00162866	664	.00150602	714	.00140056	764	.00130890
615	.00162602	665	.00150376	715	.00139860	765	.00130719
616	.00162338	666	.00150150	716	.00139665	766	.00130548
617	.00162075	667	.00149925	717	.00139470	767	.00130378
618	.00161812	668	.00149701	718	.00139276	768	.00130208
619	.00161551	669	.00149477	719	.00139082	769	.00130039
620	.00161290	670	.00149254	720	.00138889	770	.00129870
621	.00161031	671	.00149031	721	.00138696	771	.001 29702
622	.00160772	672	.00148809	722	.00138504	772	.00129534
623	.00160514	673	.00148588	723	.00138313	773	.001 29366
624	.00160256	674	.00148368	724	.00138121	774	.00129199
625	.00160000	675	.00148148	725	.00137931	775	.00129032
626	.00159744	676	.00147929	726	.00137741	776	.001 28866
627 628	.00159490	677 678	.00147710	727	.00137552	777	.00128535
	.00159236	679	.00147493	729	.00137303		.00128333
629 630	.00158982	680	.00147059	730	.0013/1/4	779 780	.00128205
631	.00158479	681	.00146843	731	.00136799	781	.00128041
632	.00158228	682	.00146628	732	.00136612	782	.00127877
633	.00157978	683	.00146413	733	.00136426	783	.00127714
634	.00157729	684	.00146199	734	.00136240	784	.00127551
635	.00157480	685	.00145985	735	.00136054	785	.00127388
636	.00157233	686	.00145773	736	.00135870	786	.00127226
637	.00156986	687	.00145560	737	.00135685	787	.00127065
638	.00156740	688	.00145349	738	.00135501	788	.00126904
639	.00156494	680	.00145137	739	.00135318	789	.00126743
640	.00156250	690	.00144927	740	.00135135	790	.00126582
641	.00156006	601	.00144718	741	.00134953	791	.00126422
642	.00155763	692	.00144509	7.12	.00134771	792	.001 26263
643	.00155521	693	.00144300	743	.00134589	793	.00126103
644	.00155279	694	.00144092	744	.00134409	794	.00125945
645	.00155039	695	.00143885	745	.00134228	795	.001 25786
646	.00154799	696	.00143678	746	.00134048	796	.00125628
647	.00154559	697	.00143472	747	.00133869	797	.00125470
648	.00154321	698	.00143266	748	.00133690	798	.00125313
649	.00154083	699	.00143061	749	.00133511	799	.00125156
650	.00153846	700	.00142857	750	.00133333	800	.00125000
	1	lł	1	11	1	11	1

RECIPROCALS OF NUMBERS FROM I TO 1000

No.	Reciprocal	No.	Reciprocal	No.	Reciprocal	No.	Reciprocal
	0	851		1	00		
801	.00124844		.00117509	901	.00110088	951	.00105152
802	.00124688	852	.00117371	902	.00110865	952	.00105042
803	.00124533	853	.00117233	903	.00110742	953	.00104032
804	.001 24378	854	.00117096	904	.00110619	954	.00104822
805	.00124224	855	.00116959	903	.00110497	955	.00104712
806	.00124069	856	.00116822	906	.00110375	956	.00104602
807	.00123016	857	.00116686	907	.00110254	957	.00101493
808	.00123762	858	.00116550	908	.00110132	958	.00101384
800	.00123600	859	.00116414	900	11001100.	959	.00104275
810	.00123457	860	.00116279	910	.00109890	960	.00104167
811	.00123305	861	.001161.14	911	.00109769	961	.00101058
812	.00123153	862	.00116009	912	.00100649	962	.00103950
813	.00123001	863	.00115875	913	.00100520	963	.00103842
814	.00122850	864	.00115741	914	.00100100	964	.00103734
815	.00122699	865	.00115607	915	.00100200	965	.00103627
816	.00122549	866	.00115473	916	.00100170	066	.00103520
817	.00122399	867	.00115340	917	.00100051	967	.00103413
818	.00122240	868	.00115207	918	.00108032	968	.00103306
819	.00122100	860	.00115075	919	.00108811	969	.00103100
820	.00122100	870	.00114042	0.20	.00108614	970	,00103199
	1			920		9,0	
821	.00121803	871	.00114811	921	.00108578	971	.00102987
822	.00121654	872	.00114679	922	.00108460	972	.00102881
823	.00121507	873	.00114547	923	.00108342	973	.00102775
824	.00121359	874	.00114116	924	.00108225	974	.00102669
825	.00121212	87.5	.00114286	925	.00108108	97.5	.00102564
826	.00121065	876	.00114155	926	.00107001	976	.00102459
827	.00120010	877	.00114025	927	.00107875	977	.00102354
828	.00120773	878	.00113895	928	.00107759	978	.00102250
829	.001 20627	879	.00113766	929	.00107643	979	.00102145
830	.00120482	886	.00113636	930	.00107527	980	.00102041
831	.00120337	881	.00113507	931	.00107411	981	.00101037
832	.00120102	882	.00113379	932	.00107296	982	.00101833
833	.00120048	883	.00113250	933	.00107181	983	.00101729
834	.00110004	884	.00113122	934	.00107066	984	.00101626
835	.00110760	885	.00112004	935	.00106052	085	.00101523
836	.00110617	886	.00112867	936	.00106838	986	.00101420
837	.00110474	887	.00112740	937	.00106724	987	.00101317
838	.00119332	888	.00112613	938	.00100610	988	,00101215
839	.00119332	880	.00112486	939	.00100010	989	.00101112
840	.00119048	890	.00112360	939	.00106383	990	.00101010
841	.00118906	801	.00112233	941	.00106270	991	.00100008
842	.00118765	802	.00112108	942	.00106157	992	.00100806
843	.00118621	893	.00111082	942	.00106011	993	.00100705
844	.00118483	894	.00111857	943	.00105032	993	.00100601
845	.00118343	895	.00111732	945	.00105820	995	.00100502
846	.00118203	896	.00111607	946	.00105708	993	,00100402
847	.00118064	897	.00111483	947	.00105597	997	.00100301
848	.00117024	898	.00111359	947	.00105397	997	.00100301
849	.00117924						.00100100
850	.00117/80	899	.00111235	949 950	.00105374	999	.00100100
	<u> </u>	11	<u> </u>	11	<u> </u>	11	

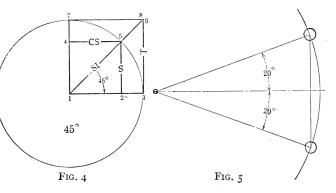
SHOP TRIGONOMETRY

The laying out of angles is sometimes difficult by ordinary methods and a little knowledge of shop "trig" is very useful and much easier than as though we called it by its full name.



It is really a system of constants or multipliers based on the fact that there are always fixed proportions between the sides and angles of triangles and other figures. Fig. 1 shows a 30-degree angle with 1, 2 and 3-inch arcs, 1 c, 2 f, and 3 i. It will be found that every similar measurement is in exact proportion to the radius, thus 2d is exactly twice the length of 1 a, and h i is just three times b c. So, if we know the distance a c for a 1-inch radius for any angle, a similar distance as g i for the same angle, will be in exact proportion to the radius of the circle to one, which is the base. All these parts are named as shown in Figs. 2, 3, and 4.

The exact proportion of all the various parts have been figured for each part of a degree that is likely to be needed in ordinary work, and these figures are given in the tables which follow. These numbers are simply multipliers or constants for a radius of one, and for any other radius we multiply the numbers given by the radius we are using. These tables form the most accurate means of calculating many problems as will be shown. These constants can represent one of anything, inches, feet, meters, or miles, and the answer will be in the same unit. In tool work they are usually in inches, but the relation is the same regardless of the unit.



Angle is Always Taken each Side of the Center Line as

Lines 1-3 and 1-7 are called radius of the circle.
1-2 is called cosine of the angle.
4-5 is always the same as cosine of the angle.
2-3 is called the versed sine of the angle.
4-7 " " co-versed sine of the angle.
2-5 " " sine " " "
3-6 " " tangent " "

7-8 " " co-tangent " " [-6 " " secant " " [-8 " " co-secant " "

The names always refer to the angle on one side of the center line and not to the total or included angle. In dealing with a 60-degree thread we divide this by a center line and call the angle 30 degrees in all our calculations. Everything is based on the radius of a circle, and a r radius is used as this base. Perhaps the three most important parts are the sine, the tangent, and the secant, these being 2-5, 3-6, and 1-6 in all three of the figures. From this it will be seen that the sine is half the chord or the distance from the radius to the horizontal. The tangent 3-6 is the distance from

the horizontal radius to an extension of the radius at the angle given. The secant is the distance along the radius from the center to the tangent. From 2 to 3 is called the *versed sine*, and is the distance

from the center of the chord to the outer circle.

The angle considered in this work is always less than 90 degrees, and the angle between the angle used and 90 degrees, or the angle which is necessary to complete this to 90 degrees is called the complementary angle. In the first case the complementary angle is 60 degrees, in the second case 45 degrees, and in the third case 30 degrees. The co-sine is the distance 4-5, the co-tangent is 7-8, the co-secant is 1-8, and the co-versed sine is 4-7 in all three examples. In the 45-degree angle it will be seen that the various parts are alike in both angles, as the sine is the same as the cosine, while the sine of the angle of 30 degrees is the same as the cosine of the angle of 60 degrees. These facts will be borne out by the tables and can be seen by studying the diagrams or by making any calculation and then proving it as near as may be on the drawing board.

All this is interesting, but unless it is useful it has no value to the practical man, so we will see where it can be used to advantage in

saving time and labor.

Perhaps the easiest application is in finding the depth of a V-thread without making any figures. The angle is 60 degrees or 30 degrees each side of the center line. The pitch is 1 inch so that each side is also an inch, and so the radius is an inch, the depth of the thread is the distance 1-2 or 4-5, and is the cosine of the angle. Looking in the table for the cosine of the angle of 30 degrees we find 0.86603 inch. If the radius is 1 this gives us the depth directly as 0.86603 inch. If the radius was 2 inches we would multiply by 2, or if it was ½ inch, divide by 2 and get the exact depth with almost no figuring. Suppose, on the other hand, that the thread was one inch deep and we want to find the length of one side, the angle remaining the same as before. In this case we have the depth which is the line 1-3, and we wish to find 1-6 which is the secant, so we look at the table again and find the secant of 30 degrees to be 1.1547 inches as the length of the side.

Suppose you have a square bar $2\frac{1}{2}$ inches on each side, what is the distance across the corners? Looking at the second example we see that the side of the square bar is represented by line r-3 and the corner distance by the secant r-6 so we look for the secant of 45 degrees (because we know that half the 90 degree angle of a square bar must be 45 degrees) and find r.4142 which would be the distance if the bar was one inch square, so we multiply r.4142 by $2\frac{1}{2}$ and get 3.5355 inches as the distance across the corners, and can know that this is closer than we can measure, and is not a guess by any means.

Reversing this we can find the side of a square that can be milled out of a round bar, such as the end of a reamer or tap. What square can we make on a 2-inch round reamer shank? The diameter of the bar is the radius as 1-5 and the angle 45 degrees as before, half the side of the square will be the sine 2-5, which the table shows to be 0.70711, and as this is half the chord which makes the flat across the bar, we multiply this by 2 and get 1.41422 inches as the distance across the flats for a reamer shank of this size.

Suppose we have a bar of $1\frac{1}{2} \times \frac{3}{4}$ -inch steel and want to find the distance across the corners, and the angle it will make with the base. The $1\frac{1}{2}$ -inch side is the radius, the diagonal is the secant, and the $\frac{3}{4}$ -inch side is the tangent of the angle. Reducing these to a basis of one inch we have a bar 1 inch by $\frac{1}{2}$ inch and the $\frac{1}{2}$ inch is the tangent of the angle. Looking in the table we find this to be almost exactly the tangent of 26 degrees and 34 minutes. With this angle

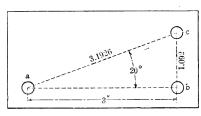
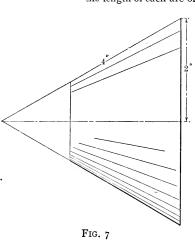


Fig. 6

the circumference by dividing 360 degrees by the number of divisions. but what we want is to find the chord or the distance from one point to the next in a straight line as a pair of dividers would step it off. First divide 360 by the number of divisions - say 9 - and get 40 degrees in each part. Fig. 5 shows this and we want the distance shown or the chord of the angle. This equals twice the sine of half the angle. Half the angle is 20 degrees and the sine for this is .342. Twice this or 0.684 is the chord of the 40-degree angle for every inch

the secant or diagonal is 1.1180 for a radius of 1 inch and 1½ times this gives 1.6770 as the distance across corners.

A very practical use for this kind of calculation is in spacing bolt holes or otherwise dividing a circle into any number of equal parts. It is easy enough to get the length of each arc of



of radius. If the circle is 14 inches in diameter the distance between the holes will be 7 times 0.684 or 4.788 inches. This is very quick and the most accurate method known.

Draftsmen often lay out jigs with the angles marked in degrees as in Fig. 6, overlooking the fact that the toolmaker has no convenient or accurate protractor for measuring the angle. Assume that a drawing shows three holes as a, b, and c, with b and c 20 degrees apart. The distance from a to b is 3 inches, what is the distance from b to c or from a to c?

As the known radius is from a to b, the distance b c is the tangent of the angle and the tangent for a one-inch radius is .36397, so for a 3-inch radius it is $3 \times .36397 = 1.09191$ inches from b to c and at right angles to it.

But we need not depend on the accuracy of the square or of the way we use it, as we can find the distance from a to c just as easily and just as accurately as we did b c. This distance is the secant, and is 1.0642 for a one-inch radius. Multiplying this by 3 = 3.1926 as the distance which can be accurately measured.

If the distance between a and c had been 3 inches, then b c would have been the sine and a b the cosine of the angle, both of which can

be easily found from the tables.

It often happens that we want to find the angle of a roller or other piece of work as Fig. 7. Always work from the center line and continue the lines to complete the angle. Every triangle has the sides and they are called the "side opposite," "side adjacent," and "hypotenuse," the first being opposite the angle, the second the base line, and the third the slant line.

The following rules are very useful in this kind of work:

(1) Sine =
$$\frac{\text{Side Opp.}}{\text{Hypot.}}$$

(6) Side Opp. = Hypot.
$$\times$$
 Sine.

(2) Cosine =
$$\frac{\text{Side Adj.}}{\text{Hypot.}}$$

(3) Tangent =
$$\frac{\text{Side Opp.}}{\text{Side Adj.}}$$

(4) Co-Tangent =
$$\frac{\text{Side Adj.}}{\text{Side Opp.}}$$

(9) Side Adj. = Co-Tan.
$$\times$$
 Side Opp.

(5) Hypot. =
$$\frac{\text{Side Opp.}}{\text{Sine.}}$$

(10) Hypot. =
$$\frac{\text{Side Adj.}}{\text{Cosine.}}$$

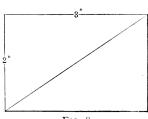


Fig. 8

If we have the dimensions shown in Fig. 7, the side opposite, and the hypotenuse, we use formula No. 1, and dividing 2 by 4 we get ½ or .5 as the sine of the angle. The table shows this to be the sine of the angle of 30 degrees, consequently this is a 30-degree angle.

If we have the side opposite and the side adjacent we use formula No. 3, and find that $\frac{2}{4} = \frac{1}{2}$ or .5 = the tangent of the angle.

The table shows this to be the tangent of 26 degrees and 34 minutes. Should it happen that we only knew the hypotenuse and the angle we use formula No. 6 and multiply $4 \times .5 = 2$, the side opposite. In the same way we can find the side adjacent by using formula No. 7. The cosine of 30 degrees is .866 and $4 \times .866 = 3.464$ inches as the side adjacent.

Having a bar of steel 2 by 3 inches, Fig. 8, what is the distance across the corners? Either formulas 3 or 4 will answer for this.

Taking No. 4 we have 2 as the side opposite, 3 as the side adjacent. Dividing 3 by 2 gives 1.5. Looking under co-tangents for this we find 1.5003 after 33 degrees 41 minutes, which is nearly the correct angle. Then look for the secant of this and find 1.2017. Multiply this by 3 and get 3.6051 as the distance across the corners.

Complete tables of sines, tangents, secants, etc., will be on pages

529 to **5**63,

USING THE TABLE OF REGULAR POLYGONS

The easiest way to lay out figures of this kind is to draw a circle and space it off, but it saves lots of time to know what spacing to use or how large a circle to draw to get a figure of the right size. Suppose we wish to lay out any regular figure, such as pentagon or five-sided figure, having sides r_2^1 inches long.

Number of Sides	Name of Figure	Diameter of Circle that will just enclose when side is I	Diameter of circle that will just go inside when side is I	Length of side where diameter of enclosure circle equals I	Length of side where inside circle equals	Angle formed by lines drawn from center to corners	Angle formed by outer sides of figures	To find Area of Figure multiply side by itself and by number in this column
3 4 5 6 7	Triangle Square Pentagon Hexagon	1.1546 1.4142 1.7012 2.	1. 1.3764 1.732 2.0766	•5 •4338	·5774 ·4815	60 51°-26′	60° 90 108 120 128 ‡	.4330 1. 1.7204 2.5980 3.6339
9 10	Octagon Nonagon Decagon Undecagon	2.6132 2.9238 3.236 3.5494	2.7474 3.0776 3.4056	.3827 .342 .309 .2817	.4142 .3639 .3247 .2936	40 36 32°-43	135 140 144 147 1 1	4.8284 6.1818 7.6942 9.3656
I 2	Dodecagon	3.8638	3.732	.2588	.2679	30	150	11.1961

Table of Regular Polygons

Looking in the third column we find "Diameter of circle that will just enclose it," and opposite pentagon we find 1.7012 as the circle that will just enclose a pentagon having a side equal to 1. This may be 1 inch or 1 anything else, so as we are dealing in inches we call it inches. As the side of the pentagon is to be $1\frac{1}{2}$ inches we multiply 1.7012 by $1\frac{1}{2}$ and get 2.5518 as the diameter of circle to draw, and take half of this or the radius 1.2759 in the compass to draw the circle. Then with $1\frac{1}{2}$ inches in the dividers we space round circle, and if the work has been carefully done it will just divide it into five equal parts. Connect these points by straight lines, and you have a pentagon with sides $1\frac{1}{2}$ inches long.

If the pentagon is to go inside a circle of given diameter, say 2 inches, look under column 5 which gives "Length of side when diameter of enclosing circle equals 1," and find 5878. Multiply by 2 as this is for a 2-inch circle, and the side will be $2 \times .5878 = 1.1756$. Take this distance in the dividers and step around the 2-inch circle. Assume that it is necessary to have a triangular end on a round shaft, how large must the shaft be to give a triangle 1.5 inches on a side?

Look in the table under column 3, and opposite triangle find 1.1546, meaning that where the side of a triangle is 1, the diameter of a circle that will just enclose it is 1.1546. As the side is 1.5, we have $1.5 \times 1.1546 = 1.7318$, the diameter of the shaft required. If the corners need not be sharp probably a shaft 1.625 would be ample.

Reversing this to find the size of a bearing that can be turned on a triangular bar of this size, look in column 4, which gives the largest circle that will go inside a triangle with a side equal to 1.

This gives .5774. Multiply this by 1.5 = .8661.

A square taper reamer is to be used which must ream 1 inch at the small end and 1.5 at the back, what size must this be across the

flats at both places?

Under column 5 find .7071 as the length of the side of a square when the diameter of the enclosing circle is 1, so this will be the side of the small end of the reamer and $1.5 \times .7071 = 1.0606$ is the side of the reamer at the large end.

FINDING THE RADIUS WITHOUT THE CENTER

It sometimes happens in measuring up a machine that we need to know the radius of curves when the center is not accessible. Three such cases are shown in Figs. 9, 10, and 11, the first two being a machine and the last a broken pulley. In Fig. 9 the rule is short enough to go in the curve while in Fig. 10 it has one end touching and the other across the sides. It makes no difference which is used so long as the distances are measured correctly, the short distance or versed sine being taken at the exact center of the chord and at right angles to it. It is easier figuring when the chord or the hight are even inches, so in measuring slip the rule until one or the other comes even; sometimes it is better to make the hight come I inch and let the chord go as it will, while at others the reverse may be true. The rule for finding the diameter is: Square half the chord, add to this the square of the hight, and divide the whole thing by the hight.

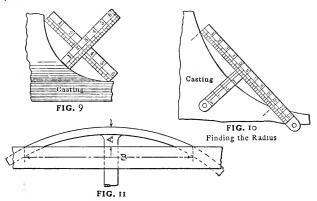
If the chord is 6 inches, as in Fig. 9, and the hight 11 inches we have

$$\frac{\frac{1}{2} \text{ chord}^2 + \text{hight}^2}{\text{hight}} = \frac{3^2 + \text{I}\frac{1}{2}^2}{\text{I}\frac{1}{2}} = \frac{9 + 2\frac{1}{4}}{\text{I}\frac{1}{2}} = \frac{\text{II}\frac{1}{4}}{\text{I}\frac{1}{2}} = 7\frac{1}{2} \text{ inches.}$$

Or as shown in Fig. 10 the chord is 10 inches and the hight 1 inch, then the figures are

$$\frac{5^2 + 1^2}{1} = \frac{25 + 1}{1} = 26$$
 inches.

In Fig. 11 we have a piece of a broken pulley, and find the chord B to be 24 inches, and the hight A to be 2 inches. This becomes $\frac{12^2 + 2^2}{2} = \frac{144 + 4}{2} = \frac{148}{2} = 74$, so that the diameter of the pulley is 74 inches.



Finding the Radius without Center

PROPERTIES OF REGULAR FIGURES

The Circle

A circle is a continuous curved line having every point at an equal distance from the center.

Its perimeter or circumference is always 3.14159265359 times the diameter, although 3.1416 is generally used and 3½ is a very close approximation.

Area equals the diameter squared × .7854, or half the diameter squared × 3.1416, or half the diameter × half the circumference.

Diameter of a square having equal area = diameter of circle times .80 very nearly.

Triangle

Equilateral triangle is a regular figure having three equal sides and three equal angles of 60 degrees each.

The side equals .866 times the diameter of enclosing circle.

Distance from one side to opposite point equals the side times .366 or diameter of enclosing circle × .75 or inside circle × 1\frac{1}{3}.

Diameter of enclosing circle equals the side times 1.1546 or 1\frac{1}{3} times distance from side to point or twice inside circle.

Diameter of inside circle equals side times .5774 or $\frac{1}{2}$ the enclosing

circle.

The area equals one side multiplied by itself and by .433013. Diameter of circle having equal area equals side of triangle times .73.

The Square

A square is a figure with four equal sides and four equal angles of 90 degrees.

Its perimeter or outside surface is four times the length of one side. Area equals one side multiplied by the other which is the same as

multiplying by itself or "squaring."

Diagonal or "long diameter," or "distance across corners," equals

the side multiplied by 1.414.

Area of circle that will go inside the square equals one side multiplied by itself times .7854 or .7854 times the area of the square.

Area of circle that will just enclose the square equals diagonal multiplied by itself times .7854 or 1.27 times the area of the square. Diameter of a circle having an equal area is 1.126 or practically

1 times the side of the square.

The Hexagon

A hexagon is a regular figure with six equal sides and six equal angles of 120 degrees. It can be drawn inside a circle by spacing around with the radius of the circle.

The side equals half the diameter of enclosing circle.

The long diameter equals diameter of enclosing circle or twice the length of one side.

The short diameter equals the long diameter multiplied by .866

or 1.732 times one side.

The area equals one side multiplied by itself and by 2.5981.

The area of enclosing circle is one side multiplied by itself and

by 3.1416.

The area of an inside circle is the short diameter multiplied by

itself and by .7854.

Diameter of circle having equal area is practically .9 times long diameter.

The Octagon

An octagon is a regular figure with eight equal sides and eight equal angles of 135 degrees.

The *side* equals the long diameter multiplied by .382. The *side* equals the short diameter multiplied by .415.

The side equals the short diameter multiplied by 415.

The long diameter equals diameter of enclosing circle or one side

multiplied by 2.62.

The short diameter equals the long diameter multiplied by .93, or one side multiplied by 2.45.

The area equals one side multiplied by itself and by 4.8284. The area of enclosing circle is 1.126 times area of octagon.

The area of inside circle is 1.120 times area of octagon.

The diameter of a circle having equal area is .953 times the long diameter of the octagon.

	. ()°	1 1	0 1	1 2	20 1	3	0	1
<u>'</u>	TAN.	Co-tan.		Co-tan.		Co-tan.		Co-tan.	
0	.00000	Infinite.	.01746	57.2900	.03492	28.6363	.05241	19.0811	60
1	.00029	3437.750	.01775	56.3506	.03521	28.3994	.05270	18.9755	59
2	.00058	1718.870	.01804	55.4415 54.5613	.03550	28.1664 27.9372	.05299	18.871 1 18.7678	58
3 4	.00116	859.436	.01862	53.7086	.03579	27.7117	.05328	18.6656	57 56
5	.00145	687.549	.01891	52.8821	.03638	27.4899	.05387	18.5645	55
5 6	.00175	572.957	.01920	52.0807	.03667	27.2715	.05416	18.4645	54
7 8	.00204	491.106	.01949	51.3032	03696	27.0566	.05445	18.3655	53
9	.00233	429.718	.01978	50.5485	.03725	26.8450 26.6367	.05474	18.2677 18.1708	52
10	.00202	381.971 343-774	.02036	49.1039	.03754	26.4316	.05533	18.0750	51 50
11	.00320	312.521	.02066	48.4121	.03812	26.2296	.05562	17.9802	49
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19 20	.00553	180.932	.02298	43.5081	.04046	24.7185	.05795	17.2558	41
21	.00582	171.885	.02328	42.4335	.04075	24.5418	.05824	17.1693	40 39
22	.00640	156.259	.02386	41.9158	.04133	24.1957	.05883	16.9990	38
23	.00669	149.465	.02415	41.4106	.04162	24.0263	.05912	16.9150	37
24	.00698	143.237	.02444	40.9174	.04191	23.8593	.05941	16.8319	36
25 26	.00727	137.507	.02473	40.4358	.04220	23.6945	.05970	16.7496 16.6681	35
27	.00785	132.219	.02502	39.9655	.04250	23.5321	.05999	16.5874	34 33
28	.00814	122.774	.02560	39.0568	.04308	23.2137	.06058	16.5075	32
29	.00844	118.540	.02589	38.6177	.04337	23.0577	.06087	16.4283	31
30	.00873	114.589	.02619	38.1885	.04366	22.9038	.06116	16.3499	30
31	.00902	110.892	.02648	37.7686	.04395	22.7519	.06145	16.2722	29 28
32 33	.00000	107.426	.02677	37·3579 36.0560	.04424	22.6020 22.4541	.06175	16.1952	27
34	.00989	101.107	.02735	36.5627	.04483	22.3081	.06233	16.0435	26
35	.01018	98.2170	.02764	36.1776	.04512	22.1640	.06262	15.9687	25
36	.01047	95.4895	.02793	35.8006	.04541	22.0217	.06291	15.8945	2.1
37 38	.01076	92,9085	.02822	35.4313	.04570	21.8813	.06321	15.8211	23
3 9	.01135	88.1436	.02881	35.0695 34.7151	.04599	21.7426	.06350	15.7483	22 21
40	.01164	85.9398	.02910	34.3678	.04658	21.4704	.06408	15.6048	20
4 I	.01193	83.8435	.02939	34.0273	.04687	21.3369	.06437	15.5340	19
42	.01222	81.8470	.02968	33.6935	.04716	21.2049	.06467	15.4638	18
43 44	.01280	79.9434 78.1263	.02997	33.3662	.04745 .04774	21.0747 20.9460	.06496	15.3943	17
45	.01300	76.3900	.03055	32.7303	.04803	20.8188	.06554	15.2571	15
46	.01338	74.7292	.03084	32.4213	.04832	20.6932	.06584	15.1893	14
47	.01367	73.1390	.03114	32.1181	.04862	20.5691	.06613	15.1222	13
48 49	.01396 .01425	71.6151	.03143	31.8205	.04891	20.4465	.06642	15.0557	12
50	.01455	70.1533 68.7501	.03172	31.5284 31.2416	.04920	20.3253	.06671 .06700	14.9898	10
51	.01484	67.4019	.03230	30.0599	.04978	20.0872	.06730	14.8596	9 8
52	.01513	66.1055	.03250	30.6833	.05007	19.9702	.06750	14.7954	
53	.01542	64.8580	.03288	30.4116	.05037	19.8546	.06788	14.7317	7 6
54 55	.01571	63.6567	.03317	30.1446 20.8823	.05066	19.7403	.06817	14.6685	5
56	.01629	61.3820	.03346	29.6245	.05124	19.5156	.06876	14.5438	4
57 58	.01658	60.3058	.03405	29.3711	.05153	19.4051	.06005	14.4823	3
58	.01687	59.2659	.03434	20.1220	.05182	19.2059	.06034	14.4212	2
59 60	.01716	58.2612 57.2900	.03463	28.8771 28.6363	.05212	19.1879	.06963	14.3607	0
-			.03492		.03241		.00993		
,	Co-tan.	TAN.	Co-tan.	TAN.	Co-tan.	TAN.	Co-tan.	TAN.	′

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,		CO-TAN.		Co-tan.	TAN.	Co-tan.	TAN.	Co-tan.	
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0	.06993	14.3007	.08749	11.4301	.10510	9.51436	.12278	8.14435	
1	.07022	14.2411	.08778	11.3919		9.46141			
2	.07051	14.1821	.08807	11.3540	.10509		12338	8.10536	
3	.07080	14.1235	.08837	11.3163	.10599	9.43515	.12367		
4	.07110	14.0655	.08866	11.2789	.10628	9.40904	.12397	8.06674	
5	.07130	14.0079	.08895	11.2417	.10057	9.38307	.12426	8.04756	
6	.07108	13.9507	.08925	11.2048	.10687	9.35724	.12456	8.02848	
8	.07197	13.8940	.08954	11.1681	.10716	9.33154	.12485	8.00948	
8	.07227	13.8378	.08983	11.1316	.10746	9.30599	.12515	7.99058	
9	.07256	13.7821	.09013	11.0054	.10775	9.28058	.12544	7.97176	
10	.07285	13.7267	.09042	11.0594	.10805	9.25530	.12574	7.95302	
11	.07314	13.6719	.09071	11.0237	.10834	9.23016	.12603	7.93438	
12	.07344	13.6174	.00101	10.9882	.10863	9.20516	12633	7.91582	
13	.07373	13.5634	.09130	10.9529	.10893	9.18028	.12662	7.89734	
14	.07402	13.5008	.09159	10.9178	.10922	9.15554	.12692	7.87895	
15	.07431	13.4566	.09189	10.8829	.10052	9.13093	.12722	7.86064	
16	.07461	13.4039	.09218	10.8483	.10981	9.10646	.12751	7.84242	
17	.07490	13.3515	.09247	10.8139	.11011	9.08211	.12781	7.82428	
18	.07519	13.2996	.09277	10.7797	.11040	9.05789	.12810	7.80622	
19	.07548	13.2480	.09306	10.7457	.11070	9.03379	.12840	7.78825	
20	.07578	13.1969	.09335	10.7119	. 11000	9.00983	.12869	7.77035	
21	.07607	13.1461	.09365	10.6783	.11128	8.98598	.12899	7.75254	1
22	.07636	13.0958	.09394	10.6450	.11158	8.96227	.12929	7.73480	ĺ
23	.07665	13.0458	.09423	10.6118	.11187	8.93867	.12958	7.71715	
24	.07695	12.9962	.09453	10.5789	.11217	8.91520	.12988	7.69957	
25	.07724	12.0460	.00482	10.5462	.11246	8.89185	.13017	7.68208	İ
26	.07753	12.8981	.00511	10.5136	.11276	8.86862	.13047	7.66466	
27	.07782	12.8496	.00541	10.4813	.11305	8.84551	.13076	7.64732	
28	.07812	12.8014	.09570	10.4491	.11335	8.82252	.13106	7.63005	
20	.07841	12.7536	.00000	10.4172	.11364	8.79964	.13136	7.61287	İ
30	.07870	12.7062	.09629	10.3854	.11394	8.77689	.13165	7.59575	
31	.07899	12.6591	.00658	10.3538	.11423	8.75425	.13195	7.57872	
32	.07929	12.6124	.09688	10.3224	.11452	8.73172	.13224	7.56176	l
33	.07958	12.5660	.00717	10.2013	.11482	8.70931	.13254	7.54487	
34	.07987	12.5100	.00746	10.2602	.11511	8.68701	.13284	7.52806	1
35	.08017	12.4742	.09776	10.2204	.11541	8.66482	.13313	7.51132	ŀ
36	.08046	12.4288	.09805	10,1988	.11570	8.64275	.13343	7.49465	
37	.08075	12.3838	.09834	10.1683	.11600	8.62078	.13372	7.47806	
38	.08104	12.3390	.00864	10.1381	.11629	8.59893	.13402	7.46154	ļ
39	.08134	12.2946	.09893	10.1080	.11659	8.57718	.13432	7.44500	
40	.08163	12.2505	.00023	10.0780	.11688	8.55555	.13461	7.42871	
41	.08192	12.2067	.00952	10.0483	.11718	8.53402	.13491	7.41240	ļ
42	.08221	12.1632	.09981	10.0187	.11747	8.51259	.13521	7.39616	
43	.08251	12.1201	.10011	9.98931	.11777	8.40128	.13550	7.37999	1
44	.08280	12.0772	.10040	0.06007	.11806	8.47007	.13580	7.36339	i
45	.08300	12.0346	.10060	0.03101	.11836	8.44806	.13600	7.34786	
46	.08339	11.9923	.10000	0.00211	.11865	8.42795	.13639	7.33190	ı
47	.08368	11.0504	.10128	9.87338	.11805	8.40705	.13660	7.31600	1
48	.08307	11.0087	.10158	0.84482	.11024	8.38625	.13698	7.30018	
49	.08427	11.8673	.10187	0.81641	.11054	8.36555	.13728	7.28442	
50	.08456	11.8262	.10216	9.78817	.11983	8.34496	.13758	7.26873	1
	.08485	11.7853	.10246	9.76009	.12013	8.32446	.13787	7.25310	
51	.08514	11.7053	.10240	9.70009	.12013	8.30406	.13817	7.23754	1
52	.08544	11.7345	.10305	0.70441	.12042	8.28376	.13846	7.22204	
53	.08573	11.7045	.10305	0.67680	.12101	8.26355	.13876	7.20661	1
54	.08602	11.6248	.10354	9.64935	.12131	8.24345	.13006	7.19125	1
55 56	.08632	11.5853	10303	0.62205	.12150	8.22344	.13935	7.17594	1
	.08661	11.5053	10393	9.59490	.12100	8.20352	.13965	7.16071	
57 58	.08600	11.5401	.10452	9.56791	.12100	8.18370	.13995	7.14553	1
59	.08720		.10481	9.54106	.12240	8.16398	.14024	7.13042	1
60	.08749		.10401	9.51436	.12278	8.14435	.14054	7.11537	-
7	1-3749				l				
,	CO-TAN	TAN.	CO-TAN.	TAN.	Co-tan	TAN.	Co-TAN	TAN.	
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,	TAN.	Co-tán.		Co-tan.	TAN.	l Co-tan.	TAN.	Co-tan.	
-	TAN.	CO-TAN.	I AN.	CO-TAIN.	I A.v.	CO-TAIN	L A.A.	CO TAIN.	
0	.14054	7.11537	.15838	6.31375	.17633	5.67128	.19438	5.14455	60
- I	.14084	7.10038	.15868	6.30189	.17663	5.66165		5.13658	59
2	.14113	7.08546	.15898	6.20007	.17693	5.65205	.19498	5.12862	58
3	.14143	7.07059	.15928	6.27829	.17723	5.64248	.19529	5.12069	57 56
4	.14173	7.05579	.15958	6.26655	.17753	5.63295	.19559	5.11279 5.10490	55
5	.14232	7.04105 7.02637	.16017	6.24321	.17813	5.61307	.19619	5.00704	54
	.14262	7.01174	.16047	6.23160	.17843	5.60452	19649	5.08021	53
7 8	.14291	6.00718	.16077	6.22003	.17873	5.59511	.1968o	5.08130	52
9	.14321	6.98268	.16107	6.20851	.17903	5.58573	.19710	5.07360	51
10	.14351	6.96823	.16137	6.19703	.17933	5.57638	.19740	5.06584	50
11	.14381	6.95385	.16167	6.18559	.17963	5.56706	.19770	5.05800	49
12	.14410	6.93952	.16196	6.17419	.17993	5.55777	.19801	5.05037	48
13	.14440	6.92525	.16226	6.16283	.18023	5.54851	.19831	5.04267	47
14	.14470	6.91104	.16256	6.15151	.18053	5.53927	.19861	5.03499	46
15	14499	6.89688	.16286	6.14023	.18113	5.53007	.19891	5.02734	45
16 17	.14529 .14559	6.88278	.16316	6.11770	.18143	5.52090	.10052	5.01210	44
18	.14588	6.85475	.16376	6.10664	.18173	5.50264	.10082	5.00451	43
10	14618	6.84582	.16405	6.09552	.18203	5.49356	.20012	4.09695	41
20	.14648	6.82604	.16435	6.08444	.18233	5.48451	-20042	4.98940	40
2 I	.14678	6.81312	.16465	6.073.40	.18263	5.47548	.20073	4.08188	39
22	.14707	6.79936	.16495	6.06240	.18293	5.46648	.20103	4.97438	38
23	.14737	6.78564	.16525	6.05143	.18323	5.45751	.20133	4.96690	37
24	.14767	6.77199	.16555	6.04051	.18353	5.44857	20164	4.95945	36
25	.14796	6.75838	.16585	6.02962	.18383	5.43966	.20194	4.95201	35
26	.14826	6.74483	.16615	6.01878	.18414	5.43077	-20224	4.94460	3.4
27	.14856	6.73133	.16645	6.00797	.18444	5.42192	.20254	4.03721	33
28 20	.14886	6.71789	.16674	5.98646	.18474	5.41300	.20205	4.92984	32 31
30	.14945	6.70450	.16734	5.97576	.18534	5.39552	.20345	4.91516	30
		-	.16764	5.06510	.18564	5.38677	.20376	4.90785	20
31 32	.14975	6.66463	.16794	5.95448	.18594	5.37805	.20406	4.90763	28
33	.15034	6.65144	.16824	5.94390	.18624	5.36936	.20436	4.89330	27
34	.15064	6.63831	.16854	5.93335	.18654	5.36070	.20406	4.83605	26
35	.15094	6.62523	.16884	5.92283	.18684	5.35206	.20497	4.87882	25
36	.15124	6.61219	.16914	5.91235	.18714	5.34345	.20527	4.87152	24
37	.15153	6.59921	.16944	5.00101	.18745	5.33487	.20557	4.86444	23
38	.15183	6.58627	.16974	5.89151	.18775	5.32631	.20588	4.85727	. 22
39	.L5213	6.57339	.17004	5.88114	.18805	5.31778	.20618	4.85013	2 I 20
40	.15243	6.56055	.17033	5.87080	11	5.30928		1	}
41	.15272	6.54777	.17063	5.86051	.18865	5.30080	.20679	4.83590	18
42	.15302	6.53503	.17093	5.85024	.18895	5.29235	.20739	4.82175	17
43 44	.15332 .15362	6.52234	.17123	5.82982	.18955	5.27553	.20770	4.81471	16
45	.15391	6.49710	.17183	5.81966	.18086	5.26715	.20800	4.80769	15
46	.15421	6.48456	.17213	5.80953	.10016	5.25880	.20830	4.80068	14
47	.15451	6.47206	.17243	5.79944	.19046	5.25048	.20861	4.70370	13
48	.15481	6.45961	.17273	5.78938	.19076	5.24218	.20891	4.78673	12
49	.15511	6.44720	.17303	5.77936	.19106	5.23301	.2002I	4.77978	II
50	.15540	6.43484	.17333	5.76937	.19136	5.22566	.20052	4.77286	10
51	.15570	6.42253	.17363	5.75941	.19166	5.21744	.20982	4.76595	8
52	.15600	6.41026	•17393	5.74949	.19197	5.20925	.21013	4.75906	
53	.15630	6.39804	.17423	5.73960	-19227	5.20107	.21043	4.75219	6
54 55	.15660	6.38587	.17453	5.72974 5.71992	.19257	5.19293	.21073	4.73851	5
56	.15719	6.36165	.17513	5.71013	.19317	5.17671	.21134	4.73170	4
57	.15749	6.34961	.17543	5.70037	19347	5.16863	.21164	4.72490	3
58	.15779	6.33761	.17573	5.69064	.19378	5.16058	.21195	4.71813	2
59	.15800	6.32566	.17603	5.68094	.19408	5.15256	.21225	4.71137	1
60	.15838	6.31375	.17633	5.67128	•19438	5.14455	.21256	4.70463	a
-	Co-tan.	TAN.	Co-TAN.	TAN.	CO-TAN.	TAN.	Co-TAN.	TAN.	,
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,	TAN.	CO-TAN.		Co-TAN.	TAN.	Co-tan.		Co-tan	,
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0	.21256	4.70463	.23087	4.33148	.24933	4.01078	.26795	3.73205	60
2	.21316	4.69121	.23148	4.32001	.24904	4.00382	.26857	3.72338	59 58
3	.21347	4.68452	.23179	4.31430	.25026	3.99592	26888	3.71907	57
4	.21377	4.67786	.23200	4.30860	.25056	3.99099	.26020	3.71476	56
5	.21408	4.67121	.23240	4.30291	.25087	3.98607	.26951	3.71046	55
	.21438	4.66458	.23271	4.29724	.25118	3.98117	.26982	3.70616	54
7	.21469	4.65797	.23301	4.29159	.25149	3.97627	.27013	3.70188	53
8	.21499	4.65138	.23332	4.28595	.25180	3.97139	.27044	3.69751	52
9	.21529	4.64480	.23363	4.28032	.25211	3.96651	.27076	3.69335	51
	1 -	1	-23393	4.27471	11		11		-
11	.21500	4.63171	.23424	4.26011	.25273	3.95680	.27138	3.68485	49
12 13	.21621	4.62518	.23455	4.20352	.25304	3.95196	.27169	3.68061 3.67638	48
14	.21682	4.61210	.23485	4.25795	.25335 .25366	3.94713 3.94232	.27231	3.67217	47 46
15	.21712	4.60572	.23510	4.25239	.25300	3.93751	.27263	3.66796	45
. 16	.21743	4.50927	.23578	4.24132	.25428	3.93731	.27294	3.66376	44
17	.21773	4.59283	.23608	4.23580	-25459	3.92793	.27326	3.65957	43
18	.21804	4.58641	.23639	4.23030	.25490	3.92316	.27357	3.65538	42
19	.21834	4.58001	.23670	4.22481	.25521	3.91839	.27388	3.65121	41
20	.21864	4.57363	.23700	4.21933	.25552	3.91364	.27419	3.64705	40
21	.21895	4.56726	.23731	4.21387	.25583	3.90890	.27451	3.64280	39
22	.21925	4.56001	.23762	4.20842	.25614	3.90417	27482	3.63874	38
23	.21956	4.55458	.23793	4.20298	.25645	3.89945	.27513	3.63461	37
2.4	.21986	4.54826	.23823	4.19756	.25676	3.89474	-27545	3.63048	36
25	.22017	4.54196	.23854	4.19215	.25707	3.89004	.27576	3.62636	35
26	.22047	4.53568	.23885	4.18675	.25738	3.88536	27607	3.62224	34
27 28	.22078	4.52941	.23916	4.18137	.25769	3.88068	.27638	3.61814	33
20	.22108	4.52316	.23946	4.17600	.25800	3.87601	.27670	3.61405	32
30	.22160	4.51693	.23977	4.17064	.25831	3.86671	.27732	3.60588	31
	1 '	1	11 '	1	11 -	3.86208	11	3.60181	-
31	.22231	4.50451	.24039	4.15997	.25893	3.85745	.27764	3.59775	29 28
32 33	.22231	4.49832	.24000	4.15465	.25924	3.85284	.27826	3.59775	27
34	.22202	4.48600	.24131	4.14405	.25986	3.84824	.27858	3.58966	26
35	.22322	4.47986	.24162	4.13877	.26017	3.84364	.27880	3.58562	25
36	.22353	4.47374	.24193	4.13350	.26048	3.83906	.27920	3.58160	24
37	.22383	4.46764	.24223	4.12825	.26079	3.83449	.27952	3.57758	23
38	.22414	4.46155	.24254	4.12301	.26110	3.82992	.27983	3.57357	22
39	.22444	4.45548	.24285	4.11778	.26141	3.82537	.28015	3.56957.	21
40	.22475	4.44942	.24316	4.11256	.26172	3.82083	.28046	3.56557	20
41	.22505	4-44338	.24347	4.10736	.26203	3.81630	.28077	3.56159	10
42	.22536	4.43735	-24377	4.10216	.26235	3.81177	.28100	3.55761	18
43	.22567	4.43134	.24408	4.09699	.26266	3.80726	.28140	3.55364	17
44	.22597	4.42534	.24439	4.09182	.26297	3.80276	.28172	3.54968 3.54573	15
45 46	.22658	4.41936	.24470	4.08000	.26359	3.79827	.28234	3.54573	14
	.22689	4.40745	.24501	4.06152	.26390	3.78931	.28266	3.53785	13
47 48	.22719	4.40152	.24562	4.07127	.26421	3.78485	.28207	3.53393	12
49	.22750	4.30560	•24593	4.06616	.26452	3.78040	.28329	3.53001	11
50	.22781	4.38969	.24624	4.06107	.26483	3.77595	.28360	3.5260)	10
51	.22811	4.38381	.24655	4.05599	.26515	3.77152	.28391	3.5221)	9
52	.22842	4.37793	.24686	4.05002	.26546	3.76700	.28423	3.51829	9 8
53	.22872	4.37207	.24717	4.04586	.26577	3.76268	.28454	3.51441	7 6
54	.22903	4.36623	.24747	4.04081	.26608	3.75828	.28486	3.51053	
55	.22934	4.36040	.24778	4.03578	.26639	3.75388	.28517	3.50666	5
56	.22964	4.35459	.24809	4.03075	.26670	3.74950	.28549	3.50279	4
57 58	.22995	4.34879	.24840	4.02574	.26701	3.74512	.28580	3.49894	3 2
59	.23026	4.3.1300	.24871	4.02074	.26733	3.74075 3.73640	.28643	3.49509	1
60	.23087	4.33148	.24933	4.01078	.26795	3 73205	.28675	3.48741	٥
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1	CO-TAN.	TAN.	CO-TAN.	TAN.	CO-TAN.	TAN.	CO-TAN.	TAN.	′
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,	TAN.	Co-tan.	Tan.	Co-tan.	TAN.	Co-tan.	TAN.	Co-tan.	_
0	.28675	3.48741	.30573	3.27085	.32492	3.07768	-34433	2.90421	60
1	.28706	3.48359	.30605	3.20745	.32524	3.07464	-34465	2.90147	59
2	.28738	3.47977	.30637	3.26406	.32556	3.07160	-34498	2.89873	58
3	.28769	3.47596	.30669	3.26007	.32588	3.06857	.34530	2.89000	57
4	.28800	3.47216	.30700	3.25729 3.25392	.32653	3.00354	·34563 ·34596	2.89327 2.89055	55
5	.28864	3.46458	.30764	3.25055	.32685	3.05950	-34628	2.88783	54
	.28895	3.46080	.30796	3.24719	.32717	3.05649	.34661	2.88511	53
7 8	.28927	3.45703	.30828	3.24383	.32749	3.05349	.34693	2.88240	52
9	.28958	3.45327	.30860	3.24049	.32782	3.05049	34726	2.87970	51
10	.28990	3.44951	.30891	3.23714	32814	3.04749	-34758	2.87700	50
11	.29021	3.44576	.30923	3.23381	.32846	3.04450	·34791	2.87430	49
12	.29053	3.44202	.30955	3.23048	.32878	3.04152	-34824	2.87161	48
13	.29084	3.43829	.30987	3.22715	.32911	3.03854	-34856	2.86892	47
1.4	.29116	3.43456	.31019	3.22384	.32943	3.03556	-34889 -34922	2.86624	46
15 16	.29147	3.43084	.31051	3.21722	.32975 .33007	3.02963	-34922	2.86089	45 44
17	.20210	3.42343	.31115	3.21392	.33040	3.02607	-34987	2.85822	43
18	.20242	3.41973	.31147	3.21063	.33072	3.02372	.35019	2.85555	42
19	.29274	3.41604	.31178	3.20734	.33104	3.02077	.35052	2.85289	41
20	.29305	3.41236	.31210	3.20406	.33136	3.01783	.35085	2.85023	40
21	.29337	3.40869	.31242	3.20079	.33160	3.01489	.35117	2.84758	39
22	.29368	3.40502	-31274	3.19752	.33201	3.01196	.35150	2.84494	38
23	.29400	3.40136	.31306	3.19426	.33233	3.00903	.35183	2.84229	37
24	.29432	3.39771	.31338	3.19100	.33266	3.00611	.35216	2.83965	36
25 26	.29463	3.39406	.31370	3.18775 3.18451	.33298	3.00319	.35248 .35281	2.83702 2.83439	35 34
27	.29495	3.38679	.31402	3.18127	.33363	2.99738	35314	2.83176	33
28	.29558	3.38317	.31466	3.17804	-33395	2.99730	.35346	2.82914	32
29	.29590	3.37955	.31498	3.17481	-33427	2.99158	-35379	2.82653	31
30	.29621	3.37594	.31530	3.17159	.33460	2.98868	.35412	2.82391	30
31	.29653	3.37234	.31562	3.16838	.33492	2.98580	-35445	2.82130	29
32	.29685	3.36875	.31594	3.16517	-33524	2.98292	·35477	2.81870	28
33	.29716	3.36516	.31626	3.16197	-33557	2.98004	-35510	2.81610	27 26
34	.29748	3.36158	.31658	3.15877	.33589 .33621	2.97717	·35543 ·35576	2.81350	25
35 36	.29/50	3.35443	.31722	3.15240	.33654	2.97144	.35608	2.80833	24
37	.29843	3.35087	31754	3.14922	.33686	2.06858	-35641	2.80574	23
38	.29875	3.34732	.31786	3.14605	.33718	2.96573	.35674	2.80316	22
39	-29906	3.34377	.31818	3.14288	-33751	2.96288	-35707	2.80059	2 I
40	.29938	3.34023	.31850	3.13972	-33783	2.96004	-35740	2.79802	20
41	.29970	3.33670	.31882	3.13656	.33816	2.95721	-35772	2.79545	10
42	.30001	3.33317	.31914	3.13341	-33848	2.95437	.35805	2.79289	18
43	.30033	3.32065	.31946	3.13027	.33881	2.95155	.35838 .35871	2.79033	17 16
44 45	.30065	3.32614	.31978	3.12713	-33913 -33945	2.94590	.35904	2.78523	15
46	.30128	3.31914	.32042	3.12087	-33943	2.04300	35937	2.78260	14
47	.30160	3.31565	.32074	3.11775	.34010	2.04028	.35969	2.78014	13
48	.30192	3.31216	.32106	3.11464	.34043	2.93748	.36002	2.77761	12
49	.30224	3.30868	.32139	3.11153	-34075	2.93468	.36035	2.77507	11
50	-30255	3.30521	.32171	3.10842	.34108	2.93189	.36068	2.77254	10
51	.30287	3.30174	.32203	3.10532	-34140	2.92910	.36101	2.77002	9 8
52	.30319	3.29829	.32235	3.10223	-34173	2.92632	.36134	2.76750	
53 54	.30351	3.29483	.32267	3.09914	.34205	2.92354	.36167	2.76498	7
55	.30362	3.28705	.32331	3.09298	.34270	2.91799	.36232	2.75996	5
56	.30446	3.28452	.32363	3.08991	.34303	2.91523	.36265	2.75746	4
57	.30478	3.28109	.32396	3.08685	-34335	2.91246	.36298	2.75496	3
58	.30500	3.27767	.32428	3.08379	.34368	2.90971	.36331	2.75246	2
59 60	.30541	3.27426	.32460	3.08073 3.07768	.34400	2.90696	.36364	2.74997 2.74748	0
_	-30573	3.27085	-32492	3.07708	-34433	2.90421	-30397	2./4/40	
7	Со-тан. 7	Tan.	Со-тан. 7	Tan.	Со-тан. 7	TAN 1°	Co-tan.	TAN.	′

	2	20°	2	1°	1) 2	20	1 2	30	1
,	TAN.	Co-tan.	TAN.	Co-tan.	TAN.	Co-tan.	TAN.	Co-TAN.	,
0	.36397	2.74748	.38386	2.60500	.40403	2.47500	.42447	2.35585	60
1	.36430	2.74499	.38420	2.60283	.40436	2.47302	.42482	2.35395	59
2	.36463	2.74251	38453	2.60057	.40470	2.47095	.42516	2.35205	58
3	.36496	2.74004	.38487	2.59831	.40504	2.46888	.42551	2.35015	57
4	36520	2.73756	.38520	2.59606	.40538	2.46682	.42585	2.34825	56
5 6	.36562	2.73500	.38553	2.59381	.40572	2.46476	.42619	2.34636	55
	.36595	2.73263	.38587	2.59156	.40606	2.46270	.42654	2.34447	54
7 8	.36628	2.73017	.38620	2.58932	.40640	2.46065	.42688*	2.34258	53
	.36661	2.7277I 2.72526	.38654	2.58708	.40674	2.45860	.42722	2.34069	52
9 10	.36727	2.72281	.38721	2.58261	.40741	2.45451	.42791	2.33881	51
			11	2.58038		2.45246	.42826		50
11	.36766	2.72036	.38754	2.57815	.40775	2.45240	.42860	2.33505	49
12	.36826	2.717,92	.38821	2.57593	.40843	2.44839	.42804	2.33317	48
13	.36859	2.71305	.38854	2.57371	.40877	2.44636	.42020	2.32943	47
15	.36802	2.71062	.38888	2.57150	.40011	2.44433	.42963	2.32756	45
16	.36925	2.70819	.38921	2.56928	.40045	2.44230	.42008	2.32570	44
17	.36058	2.70577	.38955	2.56707	.40979	2.44027	.43032	2.32383	43
18	.36001	2.70335	.38988	2.56487	41013	2.43825	.43067	2.32107	42
10	.37024	2.70094	.39022	2.56266	.41047	2.43623	.43101	2.32012	41
20	-37057	2.69853	-39055	2.56046	.41081	2.43422	.43136	2.31826	40
21	.37090	2.69612	.39080	2.55827	.41115	2.43220	.43170	2.31641	39
22	.37124	2.69371	.39122	2.55608	.41149	2.43019	.43205	2.31456	38
23	.37157	2.60131	.39156	2.55389	.41183	2.42819	.43239	2.31271	37
24	.37190	2.68892	.39190	2.55170	.41217	2.42618	.43274	2.31086	36
25	.37223	2.68653	.39223	2.54952	.41251	2.42418	.43308	2.30902	35
• 26	.37256	2.68414	.39257	2.54734	.41285	2.42218	•43343	2.30718	34
27	.37289	2.58175	.39290	2.54516	.41319	2.42019	-43378	2.30534	33
28 29	·37322 ·37355	2.67937	·39324 ·39357	2.54299	.41353	2.41819	·43412 ·43447	2.30351	32 31
30	.37388	2.67462	.39337	2.53865	.41421	2.41421	.43481	2.20084	30
		2.67225	11	2.53648			II.	2.29801	1 -
31	·37422 ·37455	2.66080	-39425 -39458	2.53048	.41455 .41490	2.41223	.43516 .43550	2.29619	29 28
32 33	.37488	2.66752	-39430	2.53217	41524	2.40827	.43585	2.29437	27
34	-37521	2.66516	.39526	2.53001	.41558	2.40629	.43620	2.20254	26
35	-37554	2.66281	-39559	2.52786	.41502	2.40432	.43654	2.20073	25
36	-37588	2.66046	-39593	2.52571	.41626	2.40235	.43689	2.28891	24
37	.37621	2.65811	.39626	2.52357	.41660	2.40038	-43724	2.28710	23
3 8	.37654	2.65576	.39660	2.52142	.41694	2.39841	.43758	2.28528	22
39	.37687	2.65342	.39694	2.51929	.41728	2.39645	-43793	2.28348	21
40	.37720	2.65109	.39727	2.51715	.41763	2.39449	.43828	2.28167	20
41	-37754	2.64875	.39761	2.51502	.41797	2.39253	.43862	2.27987	19
42	.37787	2.64642	-39795	2.51289	.41831	2.30058	.43897	2.27806	18
43	.37820	2.64410	.39829	2.51076	.41865	2.38862	.43932	2.27626	17
44	.37853 .37887	2.64177	.39862	2.50864	.41899	2.38668	.43966	2.27447	16
45 46	.37920	2.63945	-39030	2.50440	.41933	2.38473	.44036	2.27207	14
47	-37953	2.63483	.39963	2.50220	.42002	2.38084	.44071	2.26000	13
48	.37986	2.63252	-30007	2.50018	.42036	2.37801	.44105	2.26730	12
49	.38020	2.63021	.40031	2.49807	.42070	2.37697	.44140	2.26552	II
50	.38053	2.62791	.40065	2.49597	.42105	2.37504	.44175	2.26374	10
51	.38086	2.62561	.40098	2.49386	.42130	2.37311	.44210	2.26106	0
52	.38120	2.62332	.40132	2.49177	.42173	2.37118	.44244	2.26018	8
53	38153	2.62103	.40166	2.48967	.42207	2.36925	.44279	2.25840	7
54	.38186	2.61874	.40200	2.48758	.42242	2.36733	.44314	2.25663	6
5.5	.38220	2.61646	40234	2.48549	.42276	2.36541	.44349	2.25486	5
56	.38253	2.61418	.40267	2.48340	.42310	2.36349	-44384	2.25309	4
57	.38286	2.61190	.40301	2.48132	.42345	2.36158	.44418	2.25132	3
58	.38320	2.60963	.40335	2.47024	.42379	2.35967	•44453	2.24950	2 1
59 60	.38353 .38386	2.60736	.40369	2.47716	.42413 .42447	2.35776 2.35585	.44488 .44523	2.24780	â
-	-30300	2.00309	-40403	4/309	-42447	33333	-44323	2.24004	-,-
′	CO-TAN. TAN.		TAN. CO-TAN. TAN.		Co-tan.	TAN.	Со-тан. Тан. 66°		
	69°								

,	2	4°	1 2	5°	11 2	6°	n 2	7°	1
	TAN.	Co-tan.	Tan.	Co-tan.	TAN.	Co-tan.	TAN.	Co-tan.	
0	·44523	2.2460.1	.46631	2.14451	.48773	2.05030	.50953	1.96261	60
I	-44558	2.24428	.46666	2.14288	.48809 .48845	2.04879	.50989	1.96120	59
2	•44593	2.24252	.46702	2.14125	.48881	2.04728	.51026	1.95979	58
3	.44627	2.24077	.46737	2.13963		2.04577	.51063	1.95838	57
4	.44662	2.23902	.46772	2.13801	.48917			1.95698	56
5 6	.44697	2.23727	.46808	2.13639	.48953 .48989	2.04276	.51136	1.95557	55
	·44732 ·44767	2.23553	.46879	2.13477	.40026	2.03975	.51173	1.95417	54
7 8	.44802	2.23204	.46014	2.13154	.40062	2.03825	.51246	1.05137	53 52
9	.44837	2.23030	.46950	2.12003	.49098	2.03675	.51283	1.94997	51
10	.44872	2.22857	.46985	2.12832	.49134	2.03526	.51319	1.94858	50
11	.44907	2.22683	.47021	2.12671	.49170	2.03376	.51356	1.9.1718	49
12	-44942	2.22510	.47056	2.12511	.49206	2.03227	.51393	1.94579	48
13	-44977	2.22337	-47092	2.12350	.49242	2.03078	.51430	1.94440	47 46
14	.45012	2.22164	47128	2.12190	.49278	2.02929	.51467	1.94301	46
15	.45047	2.21992	.47163	2.12030	.49315	2.02780	.51503	1.94162	45
16	.45082	2.21819	.47199	2.11871	.49351	2.02631	.51540	1.94023	44
17	.45117	2.21647	.47234	2.11711	.49387	2.02483	.51577	1.93885	43
18	.45152	2.21475	.47270	2.11552	-49423	2.02335	.51614	1.93746	42
19	.45187	2.21304	.47305	2.11392	+49459	2.02187	.51651	1.93608	4 T
20	.45222	2.21132	·47341	2.11233	-49495	2.02039	.51688	1.93470	40
21	.45257	2.20961	-47377	2.11075	·49532	2.01891	.51724	1.93332	39
22	.45292	2.20790	.47412	2.10916	.49568	2.01743	.51761	1.93195	38
23	.45327	2.20619	.47448	2.10758	.49604	2.01596	.51798	1.93057	37
24	.45362	2.20449	.47483	2.10600	.49640	2.01449	.51835	1.92920	36
25	·45397	2.20278	-47519	2.10442	.49677	2.01302	.51872	1.92782	35
26	·45432	2.20108	-47555	2.10284	.49713	2.01155	.51900	1.92645	34
27	.45467	2.19938	-47590	2.10126	-49749	2.01008	.51946	1.92508	33
28	-45502	2.19769	.47626	2.09969	.49786	2.00862	.51983	1.02371	32
29	·45537	2.19599	.47662	2.00811	.49822	2.00715	.52020	1.02235	31
30	·45573	,	.47698						30
31	.45608 .45643	2.19261	•47733	2.09498	.49894	2.00423	.52004	1.91962 1.91826	29 28
32	.45678	2.19092	.47769 .47805	2.09341	.49931	2.00277	.52131	1.91620	27
33 34	.45713	2.18755	.47840	2.00028	.50004	1.90986	.52205	1.91554	26
35	45748	2.18587	.47876	2.08872	.50040	1.00841	.52242	1.91418	25
36	.45784	2.18419	.47012	2.08716	.50076	1.00605	.52279	1.91282	24
37	.45810	2.18251	.47948	2.08560	.50113	1.00550	.52316	1.01147	23
38	.45854	2.18084	.47984	2.08405	.50140	1.99406	.52353	1.01012	22
39	.45880	2.17916	.48010	2.08250	.50185	1.00261	.52390	1.00876	21
40	.45924	2.17749	.48055	2.08004	.50222	1.00116	.52427	1.00741	20
41	.45060	2.17582	.48001	2.07939	.50258	1.08072	.52464	1.00607	10
42	.45995	2.17416	.48127	2.07785	.50205	1.08828	.52501	1.00472	18
43	.46030	2.17240	.48163	2.07630	.50331	1.08684	.52538	1.00337	17
44	-46065	2.17083	.48198	2.07476	.50368	1.08540	.52575	1.00203	16
45	.46101	2.16917	.48234	2.07321	.50404	1.08306	.52613	1.00069	15
46	.46136	2.16751	.48270	2.07167	.50441	1.08253	.52650	1.80035	14
47	.46171	2.16585	.48306	2.07014	.50477	1.98110	.52687	1.89801	13
48	.46206	2.16420	.48342	2.06860	.50514	1.97966	.52724	1.89667	I 2
49	.46242	2.16255	.48378	2.06706	.50550	1.07823	.52761	1.89533	ΙI
50	.46277	2.16090	.48414	2.06553	.50587	1.97680	.52798	1.89400	CI
51	.46312	2.15925	.48450	2.06400	.50623	1.97538	.52836	1.89266	9
52	.46348	2.15760	.48486	2.06247	.50660	1.97395	.52873	1.89133	8
53	.46383	2.15596	.48521	2.06094	.50696	1.97253	.52010	1.80000	7 6
54	.46418	2.15432	-48557	2.05942	.50733	1.97111	-52947	1.88867	
55 56	.46454	2.15268	.48593 .48620	2.05790	.50769 .50806	1.96969	.52984	1.88734	5
57	.46525	2.15104	.48665	2.05485	.50843	1.96685	.53022	1.88469	4
58	.46560	2.14777	.48701	2.05333	.50879	1.905544	.53096	1.88337	2
59	.46505	2.14614	.48737	2.05182	50016	1.06402	.53134	1.88205	ī
60	.46631	2.14451	.48773	2.05030	.50953	1.96261	.53171	1.88073	0
,	Co-tan.	TAN.	Co-TAN.	TAN.	Co-tan.	TAN.	Co-tan.	TAN.	
		5° TAN.	CO-TAN.	4° 1 AN.	CO-TAN.	3° TAN.	CO-TAN.	20 I AN.	
	, 0		11 0	1	0	U	ı 0	4	1

536 NATURAL TANGENTS AND CO-TANGENTS

	. 2	8° 1	. 2	9° 1	. 3	0°	3	1°	1
	TAN.	Co-tan.	TAN.	Co-tan.	TAN.	Co-tan.	TAN.	Co-tan.	
0	.53171	1.88073	.55431	1.80405	.57735	1.73205	.60086	1.66428	60
I	.53208	1.87941	-55469	1.80281	.57774 .57813	1.73089	.60126	1.66318	59
2	.53246	1.87809 1.87677	.55507	1.80158	.57813	1.72973	.60165	1.66000	58
3 4	.53320	1.87546	·55545 ·55583	1.79911	.57851	1.72741	.60245	1.65990	57 56
7	.53358	1.87415	.55621	1.79788	.57929	1.72625	.60234	1.65881	55
5 6	53395	1.87283	.55659	1.70665	.57968	1.72500	.60324	1.65772	54
7 8	-53432	1.87152	.55697	1.79542	.58007	1.72393	.60364	1.65663	53
	-53470	1.87021	-55736	1.79419	.58046	1.72278	.60403	1.65534	52
9	-53507	1.86891	.55774	1.79296	.58085	1.72163	.60443	1.05445	51
10	-53545	1.86760	.55812	1.79174	.58124	1.72047	.60483	1.65337	50
11	.53582	1.86630	.55850	1.79051	.58162	1.71932	.60522	1.65228	49
12	.53620	1.86499 1.86369	.55888	1.78929	.58201	1.71817	.60562	1.65120	48
13	.53694	1.86230	.55926	1.78685	.58270	1.71588	.60642	1.64903	47 46
15	.53732	1.86100	.56003	1.78563	.58318	1.71473	.60681	1.64795	45
16	.53769	1.85979	.56041	1.78441	.58357	1.71358	.60721	1.64687	44
17	.53807	1.85850	.56079	1.78319	.58396	1.71244	.60761	1.64579	43
18	.53844	1.85720	.56117	1.78198	-58435	1.71129	.60801	1.64471	42
19	.53882	1.85591	.56156	1.78077	.58474	1.71015	.60841	1.64363	41
20	.53920	1.85462	.56194	1.77955	.58513	1.70901	.60881	1.64256	40
21	-53957	1.85333	.56232	1.77834	.58552	1.70787	.60921	1.64148	39
22	·53995	1.85204	.56270	1.77713	.58591 .58631	1.70673	.60000	1.64041	38
23	.54032 .54070	1.84946	.56309	I.77592 I.77471	.58670	1.70560	.61040	1.63826	37 36
25	.54107	1.84818	.56385	1.77351	.58709	1.70332	.61080	1.63719	35
26	.54145	1.84689	.56424	1.77230	.58748	1.70219	.61120	1.63612	34
27	.54183	1.84561	.56462	1.77110	.58787	1.70106	.61160	1.63505	33
28	.54220	1.84433	.56500	1.76990	.58826	1.69992	.61200	1.63398	32
29	.54258	1.84305	.56539	1.76869	.58865	1.69879	.61240	1.63202	31
30	.54296	1.84177	.56577	1.76749	.58904	1.69766	.61280	1.63185	30
31	•54333	1.84049	.56616	1.76630	-58944	1.69653	.61320	1.63079	29 28
32	.54371	1.83922	.56654	1.76510	.58983	1.69541	.61360 .61400	1.62972	27
33 34	.54409 .54446	1.83667	.56731	1.76271	.50061	1.60316	.61440	1.62760	26
35	.54484	1.83540	.56760	1.76151	.59101	1.60203	.61480	1.62654	25
36	.54522	1.83413	.56808	1.76032	.59140	1.69091	61520	1.62548	24
37	.54560	1.83286	.56846	1.75913	.59179	1.68979	.61561	1.62442	23
38	.54597	1.83159	.56885	1.75794	.59218	1.68866	.61601	1.62336	22
39	.54635 .54673	1.83033	.56923	1.75675	.59258	1.68754	.61641	1.62230	2 I 20
40		-						1.62010	10°
41	.54711	1.82780	.57000	1.75437	.59336	1.68531	.61721	1.61014	18
42 43	.54786	1.82528	.57039	1.75200	.59415	1.68308	.61801	1.61808	17
44	.54824	1.82402	.57116	1.75082	.59454	1.68196	.61842	1.61703	16
45	.54862	1.82276	-57155	1.74964	.59494	1.68085	.61882	1.61598	15
46	.54900	1.82150	.57193	1.74846	-59533	1.67974	.61922	1.61493	14
47	-54938	1.82025	-57232	1.74728	-59573	1.67863	.61962	1.61388	13
48 49	·54975 ·55013	1.81899	.57271	1.74610	.59612	1.67752	.62003	1.01283	11
50	.55051	1.81640	.57348	1.74375	.59691	1.67530	.62083	1.61074	10
		1.81524				1.67419	.62124	1.60970	
51 52	.55089	1.81399	.57386	1.74257	.59730	1.67300	.62164	1.60865	9 8
53	.55165	1.81274	.57464	1.74022	.59809	1.67198	.62204	1.60761	7
54	.55203	1.81150	-57.503	1.73905	.59849	1.67088	.62245	1.60657	
55	.55241	1.81025	-57541	1.73788	.59888	1.66978	.62285	1.60553	5
56	-55279	1.80901	.57580	1.73671	.59928	1.66867	.62325	1.60449	4
57	-55317	1.80777	.57619	1.73555	.59967	1.66757	.62366	1.60345	3 2
58 59	·55355 ·55393	1.80520	.57696	1.73430	.60037	1.66538	.62446	1.60137	ī
60	.55431	1.80405	.57735	1.73205	.60086	1.66428	62487	1.60033	o
_						T	G	T	
	CO-TAN.	TAN.	CO-TAN.	TAN.	CO-TAN.	TAN.	Co-TAN.	TAN.	
	1 6	1°	1 0	U°	II 5	9	u 5	0	,

	. 3	2°	ρ 3	30] 3	4°	[] 3	5°	1
,	TAN.	Co-tan.	TAN.	Co-tan.	TAN.	Co-tan.	TAN.	CO-TAN.	′_
0	.62487	1.60033	.64941	1.53986	.67451	1.48256	.70021	1.42815	60
I	.62527	1.59930	.64982	1.53888	.67493	1.48163	.70064	1.42726	59
2	.62568	1.59826	.65023	1.53791	.67536	1.48070	.70107	1.42638	58
3	.62649	1.59620	.65106	1.53595	.67620	1.47885	.70191	1.42462	57 56
5	.62680	1.59517	.65148	1.53497	.67663	1.47792	.70238	1.42374	55
5 6	.62730	1.59414	.65189	1.53400	.67705	1.47699	.70281	1.42286	54
7 8	.62770	1.59311	.65231	1.53302	.67748	1.47607	.70325	1.42198	53
	.62811	1.59208	65272	1.53205	.67790	1.47514	.70368	1.42110	52
9	.62892	1.59105	.65314	1.53010	.67875	I.47422 I.47330	70455	1.42022	51 50
11	.62933	1.58900	.65397	1.52913	.67917	1.47238	.70499	1.41847	49
12	.62973	1.58797	.65438	1.52816	.67960	1.47146	.70542	1.41759	48
13	.63014	1.58695	.65480	1.52719	.68002	1.47053	-70586	1.41672	47
14	.63055	1.58593	.65521	1.52622	.68045	1.46962	.70629	1.41584	46
15 16	.63136	1.58388	.65604	1.52420	.68130	1.46778	.70073	1.41407	45 44
17	.63177	1.58286	.65646	1.52332	.68173	1.46686	.70760	1.41322	43
18	.63217	1.58184	.65688	1.52235	.68215	1.46595	.70804	1.41235	42
19	.63258	1.58083	.65729	1.52139	.68258	1.46503	.70848	1.41148	41
20	.63299	1.57981	.65771	1.52043	.68301	1.46411	.70891	1.41061	40
21	.63340	1.57879	.65813	1.51946	.68343	1.46320	.70935	1.40074	39
22	.63380 .63421	1.57778	.65854	1.51850	.68386	1.46229	.70979	1.40887	38
23 24	.63462	1.57575	.65938	1.51658	.68471	1.46046	.71066	1.40714	37 36
25	.63503	1.57474	.65080	1.51562	.68514	1.45955	.71110	1.40627	35
26	.63544	1.57372	.66021	1.51466	.68557	1.45864	.71154	1.40540	34
27	.63584	1.57271	.66063	1.51370	.68600	1.45773	.71198	1.40454	33
28	.63625	1.57170	.66105	1.51275	.68642	1.45682	.71242	1.40367	32
29 30	.63666	1.57069 1.56969	.66147	1.51179	.68685	1.45592	.71285	1.40281	31
31	.63748	1.56868	.66230	1.50088	.68771	1.45410	.71373	1.40100	20
32	.63789	1.56767	.66272	1.50893	.68814	1.45320	.71417	1.40022	28
33	.63830	1.56667	.66314	1.50797	.68857	1.45229	.71461	1.39936	27
34	.63371	1.56566	.66356	1.50702	.68900	1.45139	.71505	1.39850	26
35 36	.63912	1.56466	.66398	1.50607	.68942 .68985	1.45049	.71549 .71593	1.30764	25
37	.63994	1.56265	.66482	1.50417	.60028	1.44868	.71637	1.39679	24
38	.64035	1.56165	.66524	1.50322	.60071	1.44778	.71681	1.39507	22
39	.64076	1.56065	.66566	1.50228	.69114	1.44688	.71725	1.39421	21
40	.64117	1.55966	.66608	1.50133	.69157	1.44598	.71769	1.39336	20
41	.64158	1.55866	.66650	1.50038	.69200	1.44508	.71813	1.39250	19
42	.64199	1.55766	66692	1.49944	.69243	1.44418	.71857	1.30165	18
43	.64240	1.55666	.66734	1.49849	.69286	1.44329	.71901 .71946	1.39079	17 16
44 45	.64322	1.55567	.66776	1.49755 1.49661	.69329	1.44239 1.44149	.71940	1.38994	15
46	.64363	1.55368	.66860	1.49566	.60416	1.44060	.72034	1.38824	14
47	.64404	1.55269	.66902	1.49472	.69459	1.43970	.72078	1.38738	13
48	.64446	1.55170	.66944	1.49378	.69502	1.43881	.72122	1.38653	12
49 50	.64487	1.55071	.66086	1.49284	.60545	1.43792	.72166	1.38568 1.38484	10
51	.64560	1.54972	.67071	1.40007	.60631	1.43703	.72255	1.38399	
52	.64610	1.54774	.67113	1.40003	.69675	1.43525	.72200	1.38314	8
53	.64652	1.54675	.67155	1.48900	.69718	1.43436	.72344	1.38229	7
54	.64693	1.54576	.67197	1.48816	.69761	1.43347	.72388	1.38145	
55	.64734	1.54478	.67230	1.48722	.69804	1.43258	.72432	1.38060	5
56 57	.64775	1.54379	.67282	1.48629	.69847	1.43169	.72477	1.37976	4 3
58	.64858	1.54281	.67366	1.48442	.69934	1.42002	.72565	1.37807	2
59	.64899	1.54085	.67409	1.48349	.69977	1.42903	.72610	1.37722	1
60	.64941	1.53986	.67451	1.48256	.70021	1.42815	.72054	1.37638	0
′	Co-tan.	Таņ. 7°	Co-tan. 5	TAN.	Co-tan.	TAN.	Co-tan.	TAN.	'

	30	30	37	70 11	38	80	39)°	
,		CO-TAN.		Co-tan.	TAN.	Co-tan.	TAN.	Co-tan.	,
	6	1.37638	.75355	1.32704	.78129	1.27994	.80978	1.23490	6c
0	.72654	1.37554	.75401	1.32624	.78175	1.27917	.81027	1.23416	59
2	.72743	1.37470	.75447	1.32544	.78222	1.27841	.81075	1.23343	58
3	.72788	1.37386	.75492	1.32464	.78269	1.27764	.81123	1.23270	57
4	.72832	1.37302	-75538	1.32384	.78316	1.27688	.81171	1.23196	56
5	.72877	1.37218	.75584	1.32304	.78363 .78410	1.27511	.81268	1.23050	55 54
	.72921	1.37134	.75629	1.32224	.78457	1.27458	.81316	1.22977	53
7 8	.72966	1.37050	.75721	1.32144	.78504	1.27382	.81364	1.22904	52
	.73010	1.36883	.75767	1.31984	.78551	1.27306	.81413	1.22831	5 T
0	.73100	1.36800	.75812	1.31904	.78598	1.27230	.81461	1.22758	50
11	.73144	1.36716	.75858	1.31825	.78645	1.27153	.81510	1.22685	49
12	.73189	1.36633	.75904	1.31745	.78692	1.27077	.81558	1.22612	48
13	.73234	1.36549	.75950	1.31666	.78739	1.27001	.81606	1.22539	47
14	.73278	1.36466	.75996	1.31586	.78786	1.26925	.81655 .81703	1.22467	46 45
15	73323	1.36383	.76042	1.31507	.78834	1.26849	.81752	1.22394	45
16	.73368	1.36300	.76088	1.31427	.78028	1.26698	.81800	1.22249	43
17	.73413	1.36217	.76134 .76180	1.31269	.78975	1.26622	.81849	1.22176	42
18	.73457	1.36051	.76226	1.31190	.79022	1.26546	.81898	1.22104	41
19	.73502 .73547	1.35968	.76272	1.31110	.79070	1.26471	.81946	1.22031	40
21	.73592	1.35885	.76318	1.31031	.79117	1.26395	.81995	1.21959	39
21	.73637	1.35802	.76364	1.30052	.79164	1.26319	.82044	1.21886	38
23	.73681	1.35719	.76410	1.30873	.79212	1.26244	.82092	1.21814	37
24	.73726	1.35637	.76456	1.30795	.79259	1.26169	.82141	1.21742	36
25	.7377I	1.35554	.76502	1.30716	.79306	1.26018	.82190	1.21508	3 5 3 4
26	.73816	1.35472	.76548	1.30637	.79354	1.25043	.82287	1.21526	33
27	.73861	1.35389	.76594	1.30558	.79401	1.25867	.82336	1.21454	32
28	.73906	1.35307	.76686	1.30401	.79496	1.25792	.82385	1.21382	31
29 30	.73951 .73996	1.35142	.76733	1.30323	-79544	1.25717	.82434	1.21310	30
31	.74041	1.35060	.76779	1.30244	.79591	1.25642	.82483	1.21238	29
32	.74086	1.34978	.76825	1.30166	.79030	1.25567	.82531	1.21166	28
33	.74131	1.34896	.70871	1.30087	.79686	1.25492	.82580	1.21004	27
34	.74176	1.34814	.76918	1.30000	.79734	1.25417	.82678	1.20051	25
35	.74221	1.34732	.76964	1.29931	.79781	1.25268	.82727	1.20879	24
30	.7.1267	1.34650	77010	1.20775	.79877	1.25193	.82776	1.20808	23
37	.74312 .74357	1.34487	.77103	1.20606	.70024	1.25118	.82825	1.20736	22
38 39	.74402		.77140	1.29618	.79972	1.25044	.82874	1.20665	2 I
40	-74447	1.34323	.77196	1.29541	.80020	1.24969	.82923	1.20593	20
41	.74492	1.34242	.77242	1.20463	.80067	1.24895	.82972	1.20522	19
42	.74538		.77289	1.29385	.80115		.83022	1.20451	18
43	.74583	1.34070	-77335	1.29307	.80163		.83071	1.20379	16
44	.74628	1.33008	.77382	1.20220	.80211 .80258		.83120	1.20237	15
45	.74674		.77428	1.20152	.80306		.83218	1.20166	14
46	-74719		.77475	1.280074	.80354		.83268	1.20095	13
47 48	.74764	1.33/34	.77568	1.28010	.80402		.83317	1.20024	12
49	.74855		.77615		.80450	1.24301	.83366	1.19953	11
50	.74000		.77661	1.28764	.80498	1.24227	.83415	1.19882	10
51	.74046	1.33430	.77708	1.28687	.80546		.83465	1.19811	8
52	.74991	1.33349	.77754	1.28610	.80594		.83514	1.19740	1 7
53		1.33268	.77801		.80642		.83613	1.19599	6
54			.77848		.80738		.83662		5
55			.77941		.80786		.83712	1.19457	4
56 57			.77988		.80834	1 1.23710	.83761		3
57 58	75263	1 1.32865	.78035	1.28148	.8088				2
59			.78082	1.28071	.80939				
60				1.27994	.80978	1.23490	.83910		
-,	Co-TA	N. TAN.	CO-TAN	TAN.	CO-TAI	N. TAN.	CO-TAN	TAN	1
	J. CO TAI	53° 17	11	52°	ii	51°	11	50°	1

	1 40°		ii 41° ii		4	20			
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′	TAN.	Co-tan.	TAN.	Co-tan.	TAN.	Co-tan.	TAN.	Co-tan.	_
0	.83910	1.19175	.86929	1.15037	.90040	1.11061	.03252	1.07237	60
1	.83960	1.19105	.86980	1.14000	.00093	1.10996	-9.3306	1.07174	59
2	.84000	1.19035	.87031	1.14902	.90140	1.10031	.03300	1.07112	58
3	.84050	1.18964	.87082	1.14834	.00100	1.10867	-93415	1.07049	57
4	.84108	1.18894	.87133	1.14767	.90251	1.10502 1.10737	.93469	1.06987	56
5 6	.84158	1.18824	.8718.4	1.14699	.00304	1.10737	.93524 .93578	1.00025	55
	.84208	1.18754	.87287	1.14565	.90337	1.10607	-93633	1.06862	54
7 8	.84258	1.18614	.87338	1.14408	.90463	1.10543	.93688	1.06738	53 52
	.84307 .84357	1.18544	.87389	1.14430	.00516	1.10478	-93742	1.06676	51
9	.84407	1.18474	.87441	1.14363	.90569	1.10414	-93797	1.06013	50
II	.84457	1.18404	.87492	1.1.1206	.00621	1.10340	.03852	1.06551	49
12	.84507	1.18334	.87543	1.14220	.00074	1.10285	.93906	1.00489	48
13	.84556	1.18264	.87595	1.14162	.90727	1.10220	.93961	1.06427	47
14	.84606	1.18104	.87646	1.14005	.90781	1.10156	.04016	1.06365	46
15	.84656	1.18125	.87698	1.14028	.90834	1.10091	.94071	1.06303	45
16	.84706	1.18055	.87749	1.13961	.00887	1.10027	.94125	1.06241	44
17	.84756	1.17986	.87801	1.13804	.90940	1.09963	.94180	1.06179	43
18	.84806	1.17916	.87852	1.13828	.90993	1.09800	.94235	1.06117	42
19	.84856	1.17846	.87904	1.13761	.01046	1.00834	.94290	1.06056	41
20	.84906	1.17777	.87955	1.13694	.91099	1.09770	.94345	1.05994	40
21	.84956	1'17708	.88007	1.13627	.01153	1.00706	.94400	1.05932	39
22	.85006	1.17638	.88059	1.13561	.91206	1.00542	-94455	1.05870	38
23	.85057	1.17569	.88110	1.13404	.01259	1.00578	.94510	1.05800	37
24	.85107	1.17500	.88214	1.13428	.91313	1.00514	.94505	1.05747	36 35
25	.85157	1.17430	.88265	1.13205	.01410	1.00386	.04676	1.05524	34
26	.85257	1.17202	.88317	1.13228	.91473	1.00322	.94731	1.05502	33
28	.85307	1.17223	.88369	1.13162	.01520	1.00258	.94786	1.05501	32
20	.85358	1.17154	.88421	1.13096	.91580	1.00105	.94841	1.05430	31
30	.85408	1.17085	.88473	1.13020	.91633	1.00131	.94896	1.05378	30
31	.85458	1.17016	.8852.1	1.12963	.01687	1.00067	-94952	1.05317	20
32	.85500	1.16047	.88576	1.12807	.91740	1.00003	.05007	1.05255	28
33	.85559	1.16878	.88628	1.12831	.91704	1.08940	.95062	1.05194	27
34	.85600	1.16809	.88685	1.12765	.91847	1.08876	.95118	1.05133	26
35	.85660	1.16741	.88732	1.12699	.01001	1.08813	.95173	1.05072	25
36	.85710	1.16672	.88784	1.12633	·91955	1.08749	.95229	1.05010	24
37	.85761	1.16603	.88836	1.12567	.92008	1.08686	.95284	1.04040	23
38	.85811	1.16535	.88888	1.12501	.92062	1.08622	.95340 .95395	1.04888	21
39	.85862 .85012	1.16466	.88992	1.12435	.92170	1.08496	.95451	1.04706	20
40			'			1 :	.05506	1.04705	10
41	.85963	1.16329	.89045	1.12303	.92224	1.08432	.95562	1.04/05	18
42 43	.86014 .86064	1.16261	.89297	1.12238	.92277 .92331	1.08306	.95618	1.04583	17
43	.86115	1.16192	.80201	1.121/2	.92331	1.08243	.95673	1.04522	16
44	.86166	1.16056	.89253	1.12041	.92439	1.08170	.95729	1.04461	15
46	.86216	1.15087	.89306	1.11075	.92493	1.08116	.05785	1.04401	14
47	.86267	1.15919	.89358	1.11909	-92547	1.08053	.05841	1.04340	1.3
48	.86318	1.15851	.89410	1.11844	.92601	1.07990	.95897	1.04279	12
49	.86368	1.15783	.89463	1.11778	.92655	1.07027	.95952	1.04218	11
50	.86419	1.15715	.89515	1.11713	.92700	1.07864	.96008	1.04158	
51	.86470	1.15647	.89567	1.11648	.02763	1.07801	.96064	1.04007	8
52	.86521	1.15579	.80620	1.11582	.02817	1.07738	.96120	1.04036	
53	.86572	1.15511	.89672	1.11517	.92872	1.07676	.96176	1.03076	7 6
54	.86623	1.15443	.89725	1.11452	.92926	1.07613	.96288	1.03855	5
55 56	.86674	1.15375	.89777 .89830	1.11387	.92980	1.07550	.96344	1.03794	4
57	.86776	1.15308	.89883	1.11321	.93034	1.07425	.96400	1.03734	3
58	.86827	1.15240	.89935	1.11101	.93143	1.07362	.96457	1.03674	2
50	.86878	1.15104	.80088	1.11126	.93197	1.07299	.96513	1.53613	1
60	.86929	1.15037	.90040	1.11061	.93252	1.07237	.96569	1.03553	0
-	-				0	T	Comus	TAN.	1
. ′	Co-TAN	TAN.	CO-TAN.	TAN.	CO-TAN.	TAN.	Co-tan	6c 1 AN.	
	1 4	90	11 4	:8°	11 4	11"	. 4	U	1

540 NATURAL TANGENTS AND CO-TANGENTS

	44°) (ı	440			1	44°		1
,		Co-tan.	′	'	TAN.	Co-tan.	•	'	TAN.	Co-tan.	
0	.96569	1.03553	60	21	.97756	1.02295	39	41	.98901	1.01112	19
1	.96625	1.03493	59	22	.97813	1.02236	38	42	.98958	1.01053	18
2	.96681	1.03433	58	23	.97870	1.02176	37	43	.99016	1.00994	17
3	.96738	1.03372	57	24	.97927	1.02117	36	44	.99073	1.00935	16
4	.96794	1.03312	56	25	.97984	1.02057	35	45	.99131	1.00876	15
5	.96850	1.03252	55	26	.08041	1.01998	34	46	.99189	1.00818	14
6	.96927	1.03192	54	27	.98098	1.01939	33	47	.99247	1.00759	13
7 8	.96963	1.03132	53	28	.98155	1.01879	32	48	.99304	1.00701	12
8	.97020	1 0 3 0 7 2	52	29	.98213	1.01820	31	49	.99362	1.00642	11
9	.97076	1.03012	51	30	.98270	1.01761	30	50	.99420	1.00583	CI
10	.97133	1.02052	50	31	.98327	1.01702	20	51	.99478	1.00525	9
11	.97189	1.02802	49	32	.98384	1.016.42	28	52	.99536	1.00467	8
12	.97246	1.02832	48	3.3	.08441	1.01583	27	53	-99594	1.00408	7
13	.97302	1.02772	47	34	.98499	1.01524	26	54	.99652	1.00350	6
14	-97359	1.02713	46	35	.08556	1.01465	25	5.5	.99710	1.00201	5
15	.97416	1.02653	45	36	.98613	1.01406	24	56	.99768	1.00233	4
16	.97472	1.02593	44	37	.98671	1.01347	23	57	.99826	1.00175	3 2
17	.97529	1.02533	43	38	.98728	1.01288	22	58	.99884	1.00116	2
18	.97586	1.02474	42	39	.98786	1.01229	21	59	.99942	1.00058	1
19	.97643	1.02414	41	40	.98843	1.01170	20	60	I	1	0
20	.97700	1.02355	40		1						
•	CO-TAN.		'	,	CO-TAN.	TAN.	,	,	CO-TAN.	TAN.	,
	4	5°		1	4	.5°		1	4.	5°	1

NATURAL SINES AND COSINES

	()°	1 1		()	1 1		1 ()°	
,	SINE	Cosine		′	SINE	Cosine	′		SINE	Cosine	•
0	.00000	I	60	2 I	.00611	.99998	39	41	.01193	-99993	19
1	.00029	I	59	22	.00640	.99998	38	42	.01222	-99993	18
2	.00058	I	58	23	.00669	.99998	37	43	.01251	.99992	17
3	.00087	I	57	2.4	.00698	. 9 9098	36	44	.01280	-99992	16
4	.00116	I	56	25	.00727	-99997	35	45	.01309	.99991	15
5	.00145	I	55	26	.00756	· 9 9997	34	46	.01338	.99991	14
6	.00175	1	54	27	.00785	• 9 99997	33	47	.01367	.99991	13
7	.00204	I	53	28	.00814	• 9 9997	32	48	.01396	.99990	12
8	.00233	I	52	20	.00844	.9 9996	31	49	.01425	.99990	11
9	.00262	r	51	30	.00873	.99996	30	50	.01454	.99989	10
10	.00291	I	50	31	.00902	.9 9996	29	51	.01483	.99989	9
11	.00320	.99999	49	32	.00931	.99996	28	52	.01513	.99989	8
12	.00349	.99999	48	33	.00960	.99995	27	53	.01542	.00088	7
13	.00378	.99999	47	34	.00989	-99995	26	54	.01571	.99988	5
14	.00,107	.99999	46	35	.01018	-99995	25	5.5	.01600	.99987	5
15	.00436	• 9 99999	45	36	.01047	· 9 9995	2.1	56	.01629	.99987	4
16	.00465	•99999	44	37	.01076	-99994	23	57	.01658	.99986	3
17	.00495	.99999	43	38	.01105	-99994	22	58	.01687	.99986	2
18	.00524	-99999	42	39	.01134	-99994	21	59	.01716	.99985	1
19	.00553	.99998	41	40	.01164	-99993	20	60	.01745	.99985	0
20	.00582	.99 998	40								- 85
7	COSINE	SINE	•	,	Cosine	SINE	,	,	COSINE	SINE	•
	8	9°	1 1		8	90		1	8	9°	

	, .	0	h 5	20	1 :	30	1 .	10	l
	SINE	COSINE	SINE	Cosine	SINE	Cosine	SINE	COSINE	,
0	.01745	.99985	.03490	.99939	.05234	.99863	:06076	.99756	60
1	.01774	.99984	.03519	99938	.05263	.99861	.07005	-99754	59
2	.01803	.90984	.03548	.99037 .99936	.05292	.99860 .99858	.07034	-99752	58
3	.01862	.99983	.03577	-99935	.05321	.99857	.07003	.99750 .99748	57 56
5	.01801	.00082	.03635	.99934	.05379	.99855	.07121	.99746	55
6	.01920	.99982	.03664	-99933	.05408	.99854	.07150	-99744	54
7 8	.01949	.99981	.03693	.99932	.05437	.99852	.07179	-99742	53
	.01978	.99980	.03723	.99931	.05466	.99851	.07208	-99740	52
9	.02007	.99980 .99979	.03752	.99930 .99929	.05495	.99849	.07237	.99738° .99736	51 50
	.02065	.99979	.03810	.00027	.05553	.00846	.07295	-99734	
11	.02003	.99078	.03839	.99926	.05582	.99844	.07324	.99734	49 48
13	.02123	.99977	.03868	.00025	.05611	.99842	.07353	.90729	47
14	.02152	-99977	.03897	.99924	.05640	.99841	.07382	-99727	46
15	.02181	.99976	.03926	-99923	.05660	.99839	.07411	-99725	45
16	.02211	.99976	.03955	.99922	.05698	.99838	.07440	.99723	44
17	.02240	.99975 .99974	.03984	.99921	.05727	.99836	.07469	.99721	43
10	.02208	99974	.04013	.99918	.05785	.99833	.07527	.00716	41
20	.02327	99973	.04071	.99917	.05814	.99831	.07556	.99714	40
21	.02356	.90972	.04100	.99916	.05844	.00820	.07585	.00712	30
22	.02385	.99972	.04129	.99915	.05873	.99827	.07614	.99710	38
23	.02414	.99971	.04159	.99913	.05902	.99826	.07643	.99708	37
24	02443	.99970	04188	.99912	.05931	.99824	.07672	.99705	36
25 26	.02472	.99969	.04217	.99911	.05080	.99822	.07701	.99703	35
27	.02530	.00068	.04246	.00000	.06018	.00810	.07730	.99701	34 33
28	.02560	.90067	.04304	.00007	.06047	.00817	.07788	.90696	32
29	.02589	.99966	.04333	.99906	.06076	.99815	.07817	.90694	31
30	.02618	.99966	.04362	.99905	.06105	.99813	.07846	.99692	30
31	.02647	.99965	.04391	.99904	.06134	.99812	.07875	.90689	20
32	.02676	.99964	.04.420	.99902	.06163	.99810	.07904	.99687	28
33 34	.02705	.99963	.04449	.99901	.06192	.99808	.07933	.99685	27 26
35	.02763	.99962	.04507	.99808	.06250	.99804	.07901	.90680	25
36	.02792	.99961	.04536	.99897	.06279	.99803	.08020	.99678	24
37	.02821	.99960	.04565	.99896	.06308	.99801	.080.10	.99676	23
38	.02850	-99959	.04594	.99894	.06337	-99799	.08078	.99673	22
39 40	.02879	.99959 .99958	.04623	.99893	.06366	-99797	.08107	.99671	2 I 20
	-		.04653	.00800	i	-99795	.08165	1 - 1	
41 42	.02938	.99957 .99956	.04082	.99889	.06424	.99793 .99792	.08105	.99666 .09664	18
43	.02996	99955	.04740	.99888	.06482	.99792	.08223	.99661	17
44	.03025	-99954	.04769	.99886	.06511	.99788	.08252	.99659	16
45	.03054	-99953	.04798	.99885	.06540	.99786	.08281	-90657	15
46	.03083	.99952	.04827	.99883	.06560	.99784	.08310	.99654	14
47 48	.03112	.99952 .99951	.04856	.99882	.06598	.99782	.08330	.99652	13
49	.03170	.99950	.04014	.99879	.06656	.99778	.08303	.99647	11
50	.03199	-99949	.04943	.99878	.06685	.99776	.08426	.00644	10
51	.03228	.00048	.04972	.00876	.06714	.00774	.08455	.00642	
52	.03257	-99947	.05001	.99875	.06743	.99772	.08484	.99639	8
53	.03286	99946	.05030	.99873	.06773	.99770	.08513	.99637	7
54	.03316	-99945	.05050	.99872	.06802	.99768	.08542	.99635	
55 56	.03345	-99944 -99943	.05088	.99870 .99869	.06831	.99766	.08571	.99632	5 4
57	.03403	-99943	.05146	.99867	.06880	.99762	.08620	.99627	3
57 58	.03432	.99941	.05175	.99866	.06918	.99760	.08658	.99625	2
59	.03461	.99940	.05205	.99864	.06947	.99758	.08687	.99622	1
60	-03490	-99939	.05234	.99863	.06976	.99756	.08716	.99619	0
,	Cosine 88	SINE	Cosine 8'	Sine 70	Cosine 8	SINE	COSINE 8	SINE	'

		5°		6°		7°		30	1
_	SINE	Cosine	SINE	COSINE	SINE	Cosine	SINE	Cosine	1
0	.08716	.99619	.10453	.99452	.12187	-99255	.13917	.00027	60
1	.08745	.99617	.10482	-99449	.12216	.99251	.13946	.99023	59
2	.08774	99614	.10511	.99446	.12245	.99248	.13975	.99019	58
3	.08803	.99612	.10540	-99443	.12274	-99244	.14004	.99015	57
4	.08831	.99609	.10569	-99440	.12302	.99240	.14033	.99011	56
5 6	.08860	.99607	.10597	-99437	.12331	.99237	.14061	.99006	55
	.08889	.99604	.10626	-99434	.12360	.99233	.14090	.99002	54
7 8	.08918	.99602	.10655	-99431	.12380	.99230	.14110	.98998	53
9	.08076	.99599 .99596	.10684	.99428	.12418	.99226	.14148	.98994	52
10	.09005	·99594	.10742	.99424	.12447	.99222	.141//	.98986	51 50
11	.09034	·99594		.00418	1	1	11	.98982	1
12	.00034	.99591	.10771	.00415	.12504	.99215	•14234 •14263	.98978	49 48
13	.00002	.99586	.10820	.99413	.12562	.99208	.14203	.98973	47
14	.00121	.99583	.10858	.99412	.12501	.99204	.14320	.98969	46
15	.00150	.99580	.10887	.99406	.12620	.00200	.14349	.08065	45
16	.09179	.99578	.10916	.99402	.12649	.99197	.14378	.98961	44
17	.09208	-99575	.10945	.99399	12678	.99193	.14407	.98957	43
18	.09237	-99572	.10973	.99396	.12706	.99189	.14436	.98953	42
19	.09266	-99570	.11002	-99393	.12735	.99186	.14464	.98948	41
20	.09295	.99567	.11031	.99390	.12764	.99182	•14493	.98944	40
2 I	.09324	.99564	.11060	.99386	.12793	.99178	.14522	.98940	39
22	.09353	.99562	.11089	.99383	.12822	-99175	.14551	.98936	38
23 24	.09382	.99559 .99556	.11118	.99380	.12851	.99171	.14580	.98931	37
25	.09411	·99553	.11176	.99377 .99374	.12008	.99163	.14637	.98923	36
26	.00460	.9955I	.11205	.99379	.12037	.99160	.14666	.98919	34
27	.00408	.99548	.11234	.99367	.12066	.99156	.14605	.08014	33
28	.09527	-99545	.11263	.99364	.12995	.99152	14723	.98910	32
29	.09556	-99542	.11291	.99360	.13024	.99148	.14752	.98906	31
30	.09585	-99540	.11320	∙99357	.13053	.99144	.14781	.98902	30
31	.09614	-99537	.11349	-99354	.13081	.99141	.14810	.98897	29
32	.09642	∙99534	.11378	.99351	.13110	.99137	.14838	.98893	28
33	.0967I	-99531	.11407	-99347	.13139	.99133	14867	.98889	27
34	.09700	.99528	.11436	-99344	.13168	-99129	.14896	.98884	26
35 36	.09729	.99526	.11465	.99341 .99337	.13197	.99125 .99122	.14925 .14954	.98876	25
37	.00787	.99520	.11523	·99334	.13254	.00118	.14982	.98871	23
38	.00816	.99517	.11552	.99331	.13283	.99114	.15011	.08867	22
39	.09845	.99514	.11580	.99327	.13312	.00110	.15040	.98863	21
40	.09874	.99511	.11609	+99324	•13341	.99106	.15069	.98858	20
41	.09903	.99508	.11638	.99320	.13370	.00102	.15007	.98854	19
42	.09932	.99506	.11667	.99317	.13399	.99098	.15126	.98849	18
43	.09961	.99503	.11696	.99314	.13427	.99094	.15155	.98845	17
44	.09990	.99500	.11725	.99310	•13456	.99091	.15184 .	.98841	16
45	.10019	-99497	.11754	-99307	.13485	.99087	.15212	.98836	15
46 47	.10048	-99494 -99491	.11783	.99303	13514	.99083	.15241	.98832	14
48	.10106	.99491	.11840	.99300	.13543 .13572	.99075	.15270	.08823	13
49	.10135	.99485	.11860	.99297	.13600	.99071	.15327	.98818	11
50	.10164	.99482	.11898	.99290	.13629	.99067	.15356	.98814	10
51	.10102	99479	.11927	.99286	.13658	.99063	.15385	.08800	9
52	.10221	.99476	.11956	.99283	.13687	.99059	.15414	.98805	8
53	.10250	-99473	.11985	.99279	.13716	.99055	.15442	.98800	7 6
54	.10279	-99470	.12014	.99276	.13744	.99051	.15471	.98796	
55	.10308	.99467	.12043	.99272	.13773	.99047	.15500	.98791	5
56	.10337	.99464	.12071	.99269	.13802	.99043	-15529	.98787	4
57 58	.10366	.99461	.12100	.99265	.13831	.99039	.15557	.98782 .98778	3
59	.10395	.99450 .99455	.12129	.99202	.13889	.99035	.15500	.98778	1
60	10453	.99453	.12187	.99255	.13009	.99031	.15643	.98769	o
									-
•	COSINE	SINE	Cosine 85	SINE	Cosine	SINE	Cosine	SINE	′
	84°		1 87	5°	82	١ - ١	8	-	

	1 90		ll 10	00	II 1	110		120		
•	SINE	Cosine		Cosine		Cosine		Cosine		
0	.15643	.98769	.17365	.98481	.1908.	.98163	.20701	.97815	00	
1	.15672	.98764	.17393	.98476	.19100	.98157	.20820	.97809	59	
2	.15701	.98760	.17422	.98471	.1913.	.98152	.20548	.97803	58	
3	.15730	.98755	.17451	.98466	.10167	.98146	.20577	.97797	57	
4	.15758	.98751	.17479	.98461	.1016	.98140	.20905	.97701	56	
5	.15707	.98741	.17537	.98455 .98450	.19224	.98129	.20033	97784	1.55	
1 7	.15845	.98737	.17505	98445	.10251	.08124	.20000	-97772	54	
8	.15873	.98732	.17504	.08440	.10300	.08118	.21010	97766	5.2	
9	.15902	.98728	.17623	.98435	.19338	.98112	.21047	.97700	51	
10	.15931	.98723	.17651	.98430	.19366	.98107	.21076	-97754	50	
II	.15959	.98718	.17680	.98425	.19395	.98101	.21104	.977.48	40	
12	15988	.98714	.17708	.98420	.19423	.98096	.21132	-97742	48	
13	.16017	.98700	.17737	.98414	19452	.98000	.2116 1	.97735	47	
14	.16046	.98704	.17766	.98409	.19481	.98584	-21189	.97729	46	
15	.16074	.98700	.17794	.98404	.19509	.98079 .98073	.21218	.97723	45	
17	.16132	.98695	.17852	.98399	.19536	.98067	.21240	.97717	44	
18	.16160	.98689	.17880	.98389	19595	.98061	.21303	.97705	43	
10	.16180	.08681	.17000	.08383	.19623	.08056	.21331	.07608	42	
20	.16218	.98676	-17937	.98378	.19652	.98050	-21300	.97692	40	
2 I	.16246	.08671	.17066	.98373	.10680	.08044	.21388	.07686	39	
22	.16275	.98667	.17995	08368	.19709	.08030	.21417	.97680	38	
.23	.16304	.98662	.18023	.98362	.19737	.98033	.21445	.97673	37	
24	.16333	.98657	.18052	98357	.19766	.9S027	.21474	.97667	36	
25	.16361	.98652	.18081	.98352	19794	.98021	.21502	.97661	35	
26	.16390	.98648	.18100	-98347	.19823	.98016	.21530	-97655	34	
27 28	.16419	.98643	.18138	.98341	.19851	.98010	.21559	.97648	33	
20	.16447	.98638 .98633	.18100	.98336 .98331	.1988o	.98004	.21587	.97642 .97636	32	
30	16505	.98629	,18224	.98325	.19937	.97997	.21644	.97630	31	
31	.16533	.98624	.18252	.98320	.10065	.07087	.21672		_	
32	.16562	.98619	.18281	.98325	.19905	.97981	.21701	.97623	20 28	
33	16501	.98614	.18300	.98319	.20022	-97975	.21720	.07611	27	
34	.16620	.98600	.18338	.98304	.2005I	97969	.21758	.07604	26	
35	.16648	.98604	.18367	.98299	.20079	.97963	.21786	.97598	25	
36	.16677	.98600	18395	.98294	.20108	.97958	.21814	.97592	2.4	
37	.16706	.98595	.18424	.98288	.20136	.97952	.21843	.97585	23	
38	.16734	.98595 .98585	.18452	.98283	.20165	.97946	.21871	.97579	22	
39 40	.16763	.98580	.18509	.98277 .98272	.20103	.97940	.21899	.97573	21	
	.16820				1 1	97934		.97566	20	
41 42	.16840	.98575 .98570	.18538	.98267	.20250	.97928	.21956	.97560	10	
42	.16878	.08565	18567	.98261 .98256	.20279	.07922	.21985	.97553 .97547	18	
43	.16076	.98561	.18624	.98250	.20336	.07010	.22013	.97547 .97541	16	
45	.16935	.98556	.18652	.08245	.20364	.07005	.22070	97534	15	
46	.16964	.08551	.18681	.98240	.20393	.97899	.22008	97528	14	
47	.16092	.98546	.18710	.98234	.20421	.97893	.22126	.97521	13	
48	.17021	.98541	.18738	.98229	.20450	.07887	-22155	.97515	I 2	
49	.17050	.98536	.18767	.98223	.20478	.97881	.22183	.97.508	11	
50	.17078	.93531	.18795	.98218	.20507	.97875	.22212	.97502	10	
51	.17107	.98526	.18824	.98212	.20535	.07869	.22240	.97.196	0	
52	.17136	.98521	.18852	.08207	.20563	.97863	.22268	.97480	8	
53 54	.17164	.98516	.18881	.98201	20592	.97857	.22297	.97483 .97476	7 6	
55 55	.17222	.08506	.18038	.08100	.20020	.97845	.22325	.97470	5	
56	.17250	.98501	.18067	.98185	.20677	.97839	.22382	.97463	4	
57	.17270	.98496	.18995	.08179	.20706	.97833	.22410	.97457	3	
58	.17308	.98491	.19024	.08174	.20734	.97827	.22438	97450	2	
50	.17336	.98486	.19052	.98168	,20763	.97821	.22467	.97444	I	
60	.17365	.98481	.19081	.98163	.20791	.97815	.22495	97437	٥	
-	Cosine	SINE	COSINE	SINE	COSINE	SINE	Cosine	SINE	,	
	8(10 JUNE	79	OUNE	78	20	COSINE 77	70 SINE		
. '	0(, 1	, 19	, ,		, 1			•	

,		30		40		5°		6°	
•	SINE	Cosine	SINE	Cosine	SINE	Cosine	SINE	Cosine	•
0	.22405	.97437	.24102	.97030	.25882	.96593	.27564	.06126	60
1	.22523	.97430	.24220	.97023	.25910	.96585	.27592	.96118	59
2	.22552	-97424	.24249	.97015	.25938	.96578	.27620	.96110	58
3	.22580	-97417	.24277	.97008	.25966	.96570	.27648	.96102	57
4	.22608	.97411	.24305	.97001	.25994 .26022	.96562	.27676	.96094 .96086	56
5 6	.22637	.97404	.24333 .24362	.96994 .96987	.26050	.96555 .96547	.27704 .2773I	.96078	55 54
	.22693	.97391	.24302	.96985	.26070	.96540	•27759	.96070	53
7 8	.22722	97384	.24418	.96973	.26107	.96532	.27787	.96062	52
9	.22750	.97378	.24446	.96966	.26135	96524	.27815	96054	51
10	.22778	.97371	•24474	.96959	.26163	.96517	.27843	.96046	50
11	.22807	.97365	.24503	.96952	.26191	.96509	.27871	.96037	49
12	.22835	.97358	·2453I	.96945	.26219	.96502	.27899	.96029	48
13	.22863	.97351	-24559	.96937	.26247	.96494	.27927 .27955	.96021	47 46
14	.22092	•97345 •97338	.24587	.96930 .96923	.26303	.96479	.27983	.96005	45
16	.22948	.97331	.24644	.96923	.26331	.96471	.28011	-95997	44
17	.22977	97325	.24672	.96900	.26359	.96463	.28039	.95989	43
18	.23005	.97318	.24700	.96902	.26387	.96456	.28067	.9598I	42
19	.23033	.97311	-24728	.96894	.26415	.96448	.28095	-95972	41
20	.23062	.97304	.24756	.96887	.26443	.96440	.28123	-95964	40
21	.23090	.97298	.24784	.96880	.26471	.96433	.28150	+95956	39
22	.23118	.97291	.24813	.96873 .96866	.26500 .26528	.96425 .96417	.28178	.95948 .95940	38 37
23 24	.23146	.97278	.24869	.96858	.26556	.96410	.28234	.95931	36
25	.23203	.97271	.24897	.96851	.26584	.96402	28262	•95923	35
26	.23231	.97264	.24925	.96844	.26612	.96394	.28290	.95915	34
27	.23260	.97257	.24954	.96837	.26640	.96386	.28318	.95907	33
28	.23288	.97251	.24982	.96829	.26668	.96379	.28346	.95898	32
29 30	.23316	.97244	.25010	.96822	.26696	.96371 .9636 3	-28374 -28402	.95890	30
		.97237	.25066	.96807	.26752		.28420	.95874	20
31 32	.23373 .23401	.97230	.25004	.96800	.26780	.96355	.28457	.95865	28
33	.23429	.97217	.25122	.96793	.26808	.96340	.28485	.95857	27
34	.23458	.97210	.25151	.96786	.26836	.96332	.28513	.95849	26
35	.23486	.97203	.25179	.96778	.26864	.96324	.28541	.95841	25
36	.23514	.97196	.25207	.96771	.26892	.96316	.28569	.95832	24
37 38	.23542 .23571	.97189	.25235	.96764	.26920	.96308 .96301	.28597	.95824 .95816	23
39	.23599	.97176	.25203	.96749	.26976	.96293	.28652	.95807	21
40	.23627	.97169	.25320	.96742	.27004	.96285	.2868o	-95799	20
41	.23656	.07162	.25348	.96734	.27032	.96277	.28708	.05701	10
42	.23684	.97155	.25376	.96727	.27060	.96269	.28736	.95782	18
43	.23712	.97148	.25404	.96719	.27088	.96261	.28764	-95774	17
44	.23740	.97141	.25432	.96712	.27116	.96253	.28792	.95766	16
45 46	.23769	97134	•25460	·96705	.27144	.96246	.28820	-95757	15
47	.23797	.97127	.25488	.96697 .96690	.27172	.96238	.28875	·95749 ·95740	13
48	.23853	.97113	.25545	.96682	.27228	.06222	.28903	.95732	12
49	.23882	.97106	-25573	.96675	.27256	.96214	.28931	.95724	11
50	.23910	-97100	.25601	.96667	.27284	.96206	.28959	.95715	10
51	.23938	.97093	.25629	.96660	.27312	.96198	.28987	-95707	8
52	·23966	.97086	.25657	.96653	.27340	.96190	.29015	.95698	
53	.23995	.97079	.25685	.96645	.27368	.96182	.29042	.95690	7 6
54 55	.24023	.97072 .97065	.25713 .25741	.96638 .96630	.27396	.96174 .96166	.29070	.95673	5
56	.24070	.97058	.25769	.96623	.27452	.06158	.20126	.95664	4
£7	.24108	.97051	.25798	.96615	.27480	.96150	.29154	.95656	3 2
58	.24136	.97044	.25826	.96608	.27508	.96142	.29182	.95647	
59	.24164	-97037	.25854	.96600	.27536	.96134	.29209	.95639	1
60	.24192	.97030	.25882	.96593	.27564	.96126	.29237	.95630	
•	Cosine 76	SINE	Cosine 75	SINE	Cosine 74	Sine 1°	Cosine 73	SINE	•

	1 170		1 1:	8° 1	11	19° II		20° 1	
٠		Cosine	SINE	Cosine	SINE	Cosine		Cosine	′_
0	.29237	.95630	.30902	.95106	·32557	-94552	-34202	.93969	60
1	.29265	.95622	.30929	.95097	.32584	-94542	.34229	-93959	59
2	.29293 .29321	.95605	.30957	.95088 .95079	.32612	·94533	·34257 ·34284	-93949 -93939	58
3	.29321	.95596	.31012	.05079	-32059	.94514	·34204 ·34311	.93939	57 56
5	.29376	.95588	.31040	.95061	-32694	.94504	•34339	-93919	55
6	.29404	-95579	.31068	.95052	.32722	+94495	·34366	.93909	54
7 8	.29432	.95571	.31095	.95043	-32749	-94485	•34393	.93899	53
8	.29487	.95562 -95554	.31123	.95033	.32777	.94476 .94466	·34421 ·34448	.93889	52 51
10	.29407	95545	.31178	.95015	.32832	-94457	•34475	.93869	50
11	.29543	.95536	.31206	.95006	.32859	-94447	-34503	.93859	40
12	.29571	.95528	-31233	94997	.32887	.94438	-34530	.93849	48
13	.29599	-95519	-31261	.94988	•32914	.94428	•34557	.93839	47
14	.29626	.95511	.31289	-94979	.32942	.94418	-34584	.93829	46
15	.29654 .29682	.95502	-31316	.94970 .94961	32909	.94409 .94399	-34612 -34639	.93819	45 44
16	.29002	•95493 •95485	•31344 •31372	.94952	.33024	.94399	-34666	93799	44
18	29737	.95476	-31300	94943	.33051	.94380	.34694	.93789	42
19	.29765	.95467	-31427	•94933	.33079	•94370	-34721	93779	41
20	.29793	•95459	-31454	.94924	.33106	.94361	•34748	.93769	40
21	.29821	.95450	.31482	-94915	•33134	.94351	·34775	.93759	39
22	.29849	.95441	.31510	.94906 .94897	.33161	-94342	.34803 .34830	.93748 .93738	38
23	.20004	·95433	·31537 ·31565	.94888	.33189 .33216	.94332 .94322	-34857	.93738	37 36
25	.29932	.95415	-31593	.94878	-33244	.94313	-34884	.93718	35
26	.29960	95407	.31620	.94869	-33271	.94303	-34912	.93708	34
27	.29987	.9 5398	.31648	.94860	-33298	.94293	•34939	.93698	33
28	-30015	.95389	-31675	.94851	.33326	94284	·34966	.93688 .93677	32
30	.30043	.95380 .95372	.31703 .31730	.94832	•33353 •33381	.94274 .94264	·34993 ·35021	.93667	31 30
31	.30098	.95363	.31758	.94823	.33408	94254	.35048	.03657	20
32	.30126	95354	.31786	.94814	-33436	.94245	-35075	.93647	2Š
33	.30154	95345	.31813	.94805	•33463	-94235	-35102	.93637	27
34	.30182	.95337	-31841	•94795	•33490	.94225	.35130	.93626 .93616	26
35	-30209 -30237	.95328	.31868	.94786	•33518 •33545	.94215 .94206	·35157 ·35184	.93606	25 24
37	.30265	.95319	·31923	.94768	•33573	.04106	-35211	.93596	23
38	.30292	.95301	.31951	.94758	•33600	.94186	•35239	-93585	22
39	.30320	.95293	.31979	94749	-33627	.94176	-35266	·93575	21
40	-30348	.95284	.32006	.94740	·33 ⁶ 55	-94167	•35293	.93565	20
41	.30376	.95275	-32034	.94730	•33682	-94157	-35320	-93555	18
42	.30403	.95266	.32061 .32089	.94721 .94712	•33710 •33737	.94147 .94137	·35347 ·35375	•93544 •93534	17
44	.30459	.95248	.32116	.94702	.33764	.94127	.35402	93524	16
45	.30486	.95240	.32144	.94693	•33792	.94118	-35429	93514	15
46	-30514	.95231	.32171	.94684	-33819	.94108	-35456	-93503	14
47	-30542	.95222	-32199	-94674	.33846	.94098 .94088	-35484	•93493 •93483	13
48 49	.30570	.95213 .95204	·32227 ·32254	.94665 .94656	•33874 •33901	.94078	.35511 .35538	.93472	11
50	.30625	.95195	.32282	.94646	-33929	.94068	.35565	.93462	10
51	.30653	.95186	.32309	.04637	-33956	.94058	-35592	-93452	9
52	•3068o	.95177	.32337	.94627	.33983	94049	.35619	.93441	8
53	.30708	.95168	-32364	.94618	.34011	.94039	.35647	·9343I	7
54	.30736	.95159	.32392	.94609	.34038	.94029	.35674	.93420	5
55 56	.30763	.95150 .95142	.32419	94599 94590	.34065 .34093	.94019	.35728	.93410	4
57	.30819	95133	-32474	.94580	.34120	93999	-35755	.93389	3
58	.30846	.95124	-32502	94571	-34147	·93989	-35782	-93379	2
59	-30874	.95115	.32529	-94561	•34175	-93979	.35810	.93368	I
60	.30902	.95106	.32557	-94552	-34202	.93969	-35837	-93358	
′	Cosine 7	SINE 2°	Cosine 7	SINE 1°	Cosine 7	O° ·	Cosine 6	9° Sine	′

	. 2	1°	11 2	2°	2	3°	11 2	40	1
′	SINE	Cosine	SINE	COSINE	SINE	Cosine	SINE	COSINE	1
0	.35837	.93358	.37461	.92718	.39073	.92050	.40674	.91355	6
1	.35864	.93348	.37488	.92707	.30100	.92039	.40700	.91343	1 5
2	.35891	-93337	.37515	.92697	.30127	.92028	.40727	.91331	5
3	.35918	.93327 .93316	·37542 ·37569	.92675	.39153	.92016	.40753 .40780	-91319	1 3
4	-35943	.03310	-37595	.02664	.39207	.91994	.40806	.91307	1
5 6	.36000	.93295	.37622	.02053	-30234	.01082	.40833	-91283	1
7 8	.36027	.93285	.37649	.92642	-39260	.91971	.40860	-91272	1
8	.36054	-93274	.37676	.92631	.39287	-91959	.40886	-91260	
9	.36081	.93264	-37703	.92620	-39314	.91948	-40913	.91248	1
10	.36108	-93253	-37730	.92609	·3934I	.91936	.40939	-91236	1 5
11	.36135	-93243	-37757	.92598	-39367	.91925	.40966	-91224	1
12	.36162	.93232	-37784	.92587	-39394	.91914	.40992	-91212	1
13	.36190	.93222	.37811	.92576	-3942I	.91902	.41019	-91200	1
1.4	.36217	.03211	.37865	.92554	·39448 ·39474	.01870	41045	.91188	1
15 16	.30244	.03100	.37892	.92543	.39501	.01868	.41098	.91176	Ľ
17	.36298	.93180	.37919	.92532	.39528	.91856	.41125	.91152	1
18	.36325	.93169	.37946	.92521	-39555	.91845	.41151	.01140	1
19	.36352	.93150	-37973	.92510	.39581	.91833	.41178	-91128	
20	-36379	.93148	-37999	.92499	.39608	.91822	.41204	91116	1
2 I	.36406	.93137	.38026	.92488	.39635	.91810	.41231	.91104	13
22	.36434	.93127	.38053	-92477	.39661	.91799	-41257	91092	13
23	.36461	.93116	-38o8o	.92466	.39688	.91787	.41284	.91080	13
2.1	.36488	.93106	.38107	.92455	-39715	-91775	.41310	.91068	3
25 26	-36515	.93095	.38134	.92444	·39741 ·39768	.91764	41337	.91C56	3
27	.36542	.93084	.38188	.92432 .9242I	39705	.01752	41363	.01044	1 3
28	36596	.93063	-38215	.02410	.39822	.01720	41416	.01032	3
20	.36623	.93052	.38241	.02300	.39848	.91718	41443	.01008	1
30	.36650	.93042	.38268	.92388	.39875	.91706	.41469	.90996	3
31	.36677	.93031	.38295	.92377	.39902	.91694	.41496	.90984	2
32	.36704	.93023	-38322	.92366	.39928	.91683	.41522	.90972	2
33 34	.36731 .36758	.02000	.38349	.92355 .92343	·39955 ·39982	.01660	.41549 .41575	.90960	2 2
35	.36785	.92988	.38403	.92343	.40008	.91648	.41602	.00036	2
36	.36812	.02078	.38430	.02321	.40035	01636	.41628	.00024	2
37	.36839	-92967	.38456	.92310	.40062	.91625	.41655	11000.	2
38	.36867	-92956	.38483	.02200	.42088	.91613	.41681	.90899	2
39	•36894	.92945	.38510	.92287	.40115	.91601	.41707	.90887	2
40	.36921	.92935	.38537	.92276	.40141	.91590	.41734	.90875	2
4 I	.36948	.92924	.38564	.92265	.40168	.91578	.41760	.90863	I
42	-36975	.92913	.38501	•92254.		.91566	.41787	.90851	1
43	.37002	-92902	.38617	-92243	40221	.91555	.41813	.90839 .90826	I
44 45	.37029 .37056	.02881	.38644	.02231	.40248	.01543 .01531	.41840	.00814	ī
46	.37083	.92870	.38698	.02220	.40301	.01510	.41802	.00802	ī
47	.37110	.92859	.38725	.92108	.40328	.01508	.41010	.90790	ī
48	.37137	.92849	.38752	.92186	.40355	.91496	.41945	.90778	1
49	.37164	.92838	.38778	.92175	.4038 1	.91484	.41972	.90766	1
50	.37191	.92827	-38805	.92164	.40408	.91472	.41998	.90753	1
5 I	.37218	.92816	.38832	.92152	.40434	.91461	.42024	.90741	
52	.37245	.92805	.38859	.92141	-40461	.91449	.42051	.90729	
53 54	.37272	02794	.38886	.92130	.40488	.91437	.42077	.90717	
54	·37299 ·37326	.92784	.38012	.02110	.40514 .40541	.91425	.42104	.90704	
56	·37353	.92773	.38966	.92006	.40541	.91414	.42130	.00680	2
57	.37380	.92751	.38993	.92085	.40504	.01300	.42183	.00668	
58	-37407	.92740	.39020	.92073	.40621	.91378	.42200	.90655	
59	-37434	.92729	.39046	.92062	.40647	.91366	.42235	.90643	1
60	-37461	.92718	-39073	.92050	.40674	.91355	.42262	.90631	_
•	Cosine 68	SINE	Cosine 67	SINE	Cosine 66	SINE	Cosine 6	SINE	,

	1 2	5°	1 2	60	1 2'	7°	2	8°	1
•	SINE	COSINE	SINE	Cosine	SINE	Cosine	SINE	Cosine	′_
0	.42262	.90631	.43837	.89879	·45399	.89101	.46947	.88295	60
1	.42288	.90618	.43863	.89867	·45425	.89087	.46973	.88281	59
2	.42315	.90606	.43889	.89854 .89841	·45451	.89074 .89061	.46999 .47024	.88267 .88254	58
3 4	.42341	.90594	.43916 .43942	.80828	·45477 ·45503	.89048	.47050	.88240	57 56
5	.42394	.90569	.43968	.80816	.45529	.80035	.47076	.88226	55
6	.42420	.90557	.43994	.89803	.45554	.89021	.47101	.88213	54
7 8	.42446	.90545	.44020	.89790	.45580	.80008	.47127	88199	53
	-42473	.90532	.44046	.89777	.45606	.88995 .88981	.47153	.88185	52
9	-42499 -42525	.90520	.44072 .44098	.89764 .89752	.45632	.88968	.47178	.88172	51
11	.42552	.90495	44124	.89739	.45684	.88955	.47229	.88144	49
12	.42578	.90483	.44151	.80726	45710	.88942	.47255	.88130	48
13	.42604	.90470	.44177	.89713	.45736	.88928	.47281	.88117	47
14	.42631	.90458	.44203	.89700	.45762	.88915	.47306	.88103	46
15	.42657	.90446	-44229	.89687	.45787	.88902	•47332	.88089	45
16	.42683	.00433	·44255	.8967.4 .89662	.45813	.88888 .88875	47358	.88075 .88062	44
17 18	.42709 .42736	.90421	.44281 .44307	.80640	.45865	.88862	•47383 •47409	.88048	43
10	.42762	.90396	•44333	.89636	.45891	.88848	.47434	.88034	41
20	.42788	.90383	-44359	.89623	.45917	.88835	.47460	.88020	40
21	.42815	.90371	.44385	.89610	-45942	.88822	.47486	.88006	39
22	.42841	.90358	.444II	.89597	.45968	.88808	.47511	.87993	38
23 24	.42867	.90346	·44437 ·44464	.89584 .89571	.45994 .46020	.88795 .88782	·47537 ·47562	.87979 .87965	37 36
25	.42094	.90334	.44404	.80558	.46046	.88768	.47588	.87951	35
26	.42946	.90300	.44516	.89545	.46072	.88755	.47614	.87937	34
27	.42972	.90296	.44542	.89532	46097	.88741	.47639	87923	33
28	-42999	.90284	.44568	.89519	.46123	.88728	.47665	.87909	32
29	.43025	.90271	•44594	.89506	.46149	.88715	.47690	.87896 .87882	31
30	·43051	.90259	.44620	.89493 .89480	.46175 .46201	.88701	.47716	.87868	30
31	.43077	.90246	.44646 .44672	.89460	.46226	.88674	.47741	.87854	29 28
32	.43130	.00233	.44698	.89454	.46252	.88661	·47793	.87840	27
34	.43156	.90208	.44724	.89441	.46278	.88647	.47818	.87826	26
35	.43182	.90196	-44750	.89428	.46304	.88634	.47844	.87812	25
36	.43209	.90183	.44776	.89415	.46330	.88620	.47869	.87798	24
37	·43235	.90171	.44802	.89402 .89389	.46355 .46381	.88607	.47895	.87784 .87770	23
38 39	.43261 .43287	.90158	.44828 .44854	.89376	.46407	.88580	.47946	.87756	22 2I
40	.43313	.90133	.44880	.89363	.46433	.88566	·47971	.87743	20
41	.43340	.00120	.44906	.89350	.46458	.88553	-47997	.87720	10
42	.43366	.90108	.44932	.89337	.46484	.88539	.48022	.87715	18
43	-43392	.90095	.44958	.89324	.46510	.88526	.48048	.87701	17
44	.43418	·90082	.44984	.89311	.46536	.88512	.48073	.87687	16
45 46	•43445	.90070	.45010 .45036	.89298 .89285	.46561	.88499 .88485	.48099 .48124	.87673 .87650	15
47	·43471 ·43497	.90057	.45062	.89272	.46613	.88472	.48150	.87645	13
48	.43523	.90032	.45088	.89259	.46639	.88458	.48175	.87631	12
49	·43549	.90019	.45114	.89245	.46664	.88445	.48201	.87617	11
50	·43575	.90007	.45140	.89232	.46690	.88431	.48226	.87603	10
51	43602	.89994	.45166	.89219	.46716	.88417	.48252	.87589	8
52	.43628	.89981	45192	.89206 .89193	.46742	.88404 .88390	.48277	.87575 .87561	8
53 54	.43654 .43680	.89968 .89956	.45210	.89180	.46793	.88377	.48328	.87546	7 6
5.5	.43706	.89933	.45269	.89167	.46819	.88363	.48354	.87532	5
56	.43733	.89930	.45295	.89153	.46844	.88349	.48379	.87518	4
57	•43759	.89918	.45321	89140	.46870	.88336	.48405	.87504	3
58	.43785	.89905	•45347	.89127 .89114	.46896 .46921	.88322 .88308	.48430	.87490 .87476	2 I
59 60	.43811 .43837	.89879	·45373 ·45399	.80101	.46947	.88205	.48481	.87462	0
									-
•	Cosine 64	SINE	Cosine 6:	SINE 30	COSINE 6	SINE 2°	Cosine 6	SINE 1	•

	1 29°		30)° .	3:	1° i	1 35	2° 1	
'	SINE	Cosine	SINE	Cosine	SINE	Cosine	SINE	Cosine	,
0	.48481	.87462	.50000	.86603 .86588	.51504	.85717	.52992	.84805	60
1	.48506	.87448 .87434	.50025	.86573	.51529 .51554	.85702 .85687	.53017 .53041	.84789 .84774	59 58
2	.48532 .48557	.87420	.50050	.86559	.51579	.85672	.53041	.84759	57
3 4	.48583	.87406	.50101	.86544	.51604	.85657	.53001	.84743	56
5	.48608	.87391	.50126	.86530	.51628	.85642	.53115	.84728	55
5	.48634	.87377	.50151	.86515	.51653	.85627	.53140	84712	54
7 8	.48659	.87363	.50176	.86501	.51678	.85612	.53164	.84697	53
	.48684	.87349	.50201	.86486	.51703	.85597 .85582	.53189	.84681	52
9	.48710	.87335 .87321	.50227 .50252	.86471	.51728	.85567	-53214 -53238	.84666 .84650	51 50
10	.48735			.86442		.85551			-
11	.48761 .48786	.87306 .87292	.50277	.86427	.51778 .51803	.85536	.53263	.84635 .84610	49 48
12	.48811	.87278	.50327	.86413	.51828	.85521	.53312	.84604	47
14	.48837	.87264	.50352	.86398	.51852	.85506	.53337	.84588	46
15	48862	.87250	.50377	.86384	.51877	.85491	.53361	.84573	45
16	.48888	.87235	.50403	.86369	-51902	.85476	.53386	.84557	44
17 18	.48913	.87221	.50428	.86354	.51927	.85461	·53411	.84542	43
	.48938	.87207	.50453	.86340	.51952	.85446	•53435	.84526	42
19 20	.48964 .48989	.87193 .87178	.50478	.86325 .86310	.51977	.85431 .85416	.53460 .53484	.84511 .84495	41 40
1				.86295	.52026	.85401	1	.84480	
2I 22	.40014	.87164 .87150	.50528	.86281	.52051	.85385	.53509 .53534	.84464	39 38
23	.49045	.87136	.50578	.86266	.52076	.85370	.53558	.84448	37
24	.49090	.87121	.50603	.86251	.52101	.85355	.53583	84433	36
25	.49116	.87107	.50628	.86237	.52126	.85340	.53607	.84417	35
26	.49141	.87093	.50654	.86222	.52151	.85325	.53632	.84402	34
27	.49166	.87079	.50679	.86207	.52175	.85310	.53656	.84386	33
28	.49192	.87064	.50704	.86192 .86178	.52200	.85294 .85279	.53681	.84370 .84355	32
29	.49217 .49242	.87050	.50729	.86163	•52250	.85264	.53730	.84339	31 30
30		.87021	.50734	.86148	.52275	.85249	.53754	.84324	20
31	.49268	.87007	.50804	.86133	.52275	.85234	-53754	.84308	28
32 33	.49293	.86993	.50829	.86110	.52324	.85218	.53804	.84292	27
34	49344	.86978	.50854	.86104	.52349	.85203	.53828	.84277	26
35	.49369	.86964	.50879	.86089	•52374	.85188	.53853	.84261	25
36	·49394	.86949	.50904	86074	.52399	.85173	.53877	.84245	24
37	-49419	.86935	.50929	.86059 .86045	.52423	.85157	.53902 .53926	.84230	23
38 39	·49445 ·49470	.86921	.50954	.86030	.52448	.85127	.53920	.84198	21
40	·49475	.86892	.51004	.86015	.52498	.85112	-53975	.84182	20
41	.49521	.86878	.51029	.86000	.52522	.85006	.54000	.84167	10
42	.49546	.86863	.51054	.85985	-52547	.85081	.54024	.84151	18
43	.49571	.86849	.51079	.85970	.52572	.85066	.54049	.84135	17
44	.49596	.86834	.51104	.85956	-52597	.85051	.54073	.84120	16
45	.49622	.86820	.51129	.85941	.52621	.85035	-54097	.84104	15
46	49647	.86805	.51154	.85926	.52646 52671	.85020	.54122	.84088	14
47 48	.49672	.86777	.51179	85806	.52696	.84989	.54171	.84057	12
49	-49723	.86762	.51229	.85881	.52720	.84974	.54195	.84041	II
50	.49748	.86748	.51254	.85866	.52745	.84959	.54220	.84025	10
51	-49773	.86733	.51279	.85851	.52770	.84943	.54244	.84000	9
52	.49798	.86719	.51304	.85836	-52794	.84928	.54269	.83994	8
53	.49824	.86704	.51329	.85821	.52819	.84913	-54293	.83978	7 6
54	.49849	.86690	.51354	.85806 .85792	.52844	.84897 .84882	.54317 .54342	.83962	5
55 56	49874	.86675	.51379	.85792	.52803	.84866	.54342	.83940	4
57	.49899 .49924	.86646	.51404	.85762	.52018	.84851	.54391	.83915	3
58	.49950	.86632	.51454	.85747	.52943	.84836	-54415	.83899	3 2
59	-49975	.86617	-51479	.85732	.52967	.84820	-54440	.83883	I
60	.50000	.86603	.51504	.85717	-52992	.84805	-54464	.83867	°
-	COSINE	SINE	COSINE	SINE	COSINE	SINE	COSINE	SINE	1
	F	30° 51112	5	90 3112	5	80	5	7°	ł
	1 600 11		0	00 1		ן פט ע			

	1 3	3°	11 3	4°	11 3	5°	3	60	1
,	SINE	COSINE	SINE	Cosine	SINE	COSINE	SINE	Cosine	′_
0	.54464 .54488	.83867 .83851	-55919	.82904 .82887	.57358	.81915	.58779 .58802	.80902	60
2	.54513	.83835	·55943 ·55968	.82871	·57381 ·57405	.81882	.58826	.80867	59 58
3	•54537	.83819	-55992	82855	.57429	.81865	.58849	.80850	57
4	.54561	.83804	.56016	.82839	•57453	81848	.58873	.80833	56
5	.54586	83788	.56040	.82822	.57477	.81832	.58896	.80816	55
	.54610	.83772	.56064	.82806	.57501	.81815	.58920	.80799	54
7 8	.54635	.83756	.56088	.82790	.57524	.81798	.58943	.80782	53
9	.54659	.83740	.56112	.82773 .82757	.57548	.81782	.58967	.80765	52
10	.54708	.83708	.56160	.82741	-57572 -57596	.81748	.58990	.80748	51 50
11	-54732	83692	.56184	.82724	.57619	.81731	.59037	.80713	49
12	-54756	.83676	.56208	.82708	.57643	.81714	.59061	.80696	48
13	.54781	.83660	.56232	.82692	.57667	.81698 .81681	.59084	.80679	47
14 15	.54805	.83645	.56256 .56280	.82675 .82659	.57691 .57715	.81664	.59108	.80662	46
16	.54854	.83613	.56305	.82643	.57738	.81647	.59151	80627	45 44
17	.54878	.83597	.56329	.82626	.57762	81631	.59178	.80610	43
18	.54902	.83581	.56353	.82610	.57786	.81614	.59201	.80593	42
19	-54927	.83565	.56377	82593	.57810	.81597	.59225	.80576	41
20	-54951	.83549	.56401	.82577	-57833	.81580	-59248	.80558	40
2 I 22	-54975 -54999	.83533	.56425	.82561	.57857	.81563 .81546	-59272	.80541	39
23	-55024	.83501	.56473	82528	.57904	.81530	.59295 .59318	.80524	38
24	.55048	.83485	.56497	.82511	.57928	.81513	-59342	.80480	36
25	.55072	.83469	.56521	.82495	-57952	.81496	.59365	.80472	35
26	-55097	.83453	.56545	.82478	.57976	.81479	.59389	.80455	34
27 28	.55121	.83437	.56569	.82462	-57999	.81462	-59412	.80438	33
	-55145	.83421	.56593	.82446	.58023	.81445	.59436	.80420	32
29 30	.55169	.83405	.56617	.82429	.58047	.81428	·59459 ·59482	.80403	31
31	.55218	.83373	.56665	82413	.58070	.81395	.59506	.80386	30
32	.55242	.83356	.56680	.823 0	.58118	.81378	-59529	.80351	28
33	.55266	.83340	.56713	.82363	.58141	.81361	-59552	.80334	27
34	.55291	.83324	.56736	.82347	.58165	.81344	.59576	.80316	26
35	.55315	.83308	.56760	.82330	.58189	.81327	-59599	.80299	25
36	-55339	.83292 .83276	.56784	.82314	.58212	.81310	.59622	.80282	24
37 38	.55363	.83260	.56832	.82297	.58236	.81293 .81276	.59646	.80247	23
39	.55412	.83244	.56856	.82264	.58283	.81250	.59693	.80230	21
40	.55436	.83228	.56880	.82248	.58307	.81242	.59716	.80212	20
4 I	.55460	.83212	.56904	.82231	.58330	.81225	-59739	.80195	10
42	.55484	.83195	.56928	.82214	.58354	81208	.59763	.80178	18
43 44	.55509	.83179 .83163	.56952	.82198 .82181	.58378	.81191	.59786	.80160 .80143	17
45	.55557	83147	.57000	.82165	.58401	.81174	.50832	.80125	15
46	.55581	.83131	.57024	.82148	.58449	.81140	.59856	.80108	14
47	-55605	.83115	.57047	.82132	.58472	.81123	.59879	.80001	13
48	.55630	.83098	.57071	.82115	.58496	.81106	-59902	.80073	12
49	.55654	.83082	-57095	.82098	.58519	.81089	.59926	.80056	11
50	.55678	83066	.57119	.82082	.58543	.81072	•59949	.80038	10
51 52	.55702 .55726	.83050 .83034	.57143 .57167	.82065 .82048	.58567	.81055 .81038	.59972 .59995	.80021 .80003	9 8
53	.55750	.83017	.57191	.82032	.58614	.81021	.60019	.79986	7
54	-55775	.83001	.57215	.82015	.58637	.81004	.60042	·79 9 68	7 6
55	-55799	.82985	.57238	.81999	.58661	.80987	.60065	·79951	5
56	.55823	82969	57262	81082	.58684	.80970	.60089	.79934	4
57 58	.55847 .55871	.82953 .82936	.57286	.81965 .81949	.58708 .58731	.80953 .80936	.60112	.79916 .70899	3
59	-55805	.82930	.57310 .57334	.81949	.58755	.80030	.60158	.79881	1
60	.55919	82904	.57358	.81915	.58779	.80902	.60182	.79864	ō
•	COSINE	SINE	COSINE	SINE	Cosine	Sine	Cosine	SINE	,
	56	30	55	50	54	fo	53 53	3°	

Sine											
0 .60182 .70864 .61566 .78801 .62932 .77715 .64279 .76604 .60 1 .60205 .70846 .61586 .78831 .62932 .77716 .64279 .76604 .60 2 .60228 .70829 .61012 .78765 .62977 .77078 .64323 .70567 .8 3 .60221 .70811 .61635 .78747 .63900 .77766 .64323 .70547 .8 4 .60274 .70703 .61681 .78747 .63900 .77766 .64323 .70547 .8 5 .60238 .70976 .61681 .78711 .60334 .77023 .64308 .70530 .5 6 .60321 .70981 .6172 .78711 .60334 .77023 .64308 .77631 .63 8 .60337 .70723 .61740 .78694 .63080 .77760 .64437 .76435 .5 9 .60390 .79706 .61772 .78640 .63135 .77550 .64479 .76435 .5 10 .60414 .79688 .61705 .78622 .63185 .77531 .64501 .76417 .50 11 .60437 .70671 .61818 .78604 .63180 .77531 .64501 .76417 .50 12 .60460 .70653 .61841 .78586 .63203 .77404 .6456 .76380 .48 13 .60483 .70635 .61864 .78588 .63235 .77444 .6456 .76380 .48 14 .60506 .70618 .61887 .78508 .63235 .77444 .6456 .76380 .48 15 .60520 .70600 .61900 .78532 .63271 .77438 .04500 .76322 .40 16 .60533 .70635 .61854 .78588 .63237 .77440 .64568 .76361 .47 17 .60576 .70618 .61887 .78508 .63325 .77444 .64568 .76362 .48 17 .60576 .70565 .61055 .78406 .63316 .77421 .64635 .76324 .40 18 .60500 .70647 .61908 .78324 .63211 .77438 .04602 .76323 .48 18 .60500 .70643 .61887 .78508 .63336 .77421 .64635 .76324 .40 17 .60576 .70565 .61055 .78406 .63316 .77442 .64657 .76286 .43 18 .60500 .70547 .61778 .78406 .63616 .77417 .60060 .70647 .42 19 .60648 .70641 .62046 .7844 .63380 .77341 .64635 .76024 .44 10 .6056 .70618 .61887 .7850 .63336 .77742 .64655 .76202 .40 10 .60622 .70530 .62001 .78406 .63616 .77348 .64070 .76267 .42 10 .60668 .70614 .62046 .78406 .63406 .77330 .64701 .76248 .41 10 .60378 .70618 .62044 .7844 .63380 .77714 .64635 .76202 .40 10 .60622 .70530 .62001 .78406 .63606 .77100 .64607 .76260 .43 12 .60668 .70614 .62046 .78406 .63606 .77100 .64607 .76260 .43 13 .60690 .707335 .62221 .78406 .63606 .77100 .64607 .76260 .43 14 .60700 .70735 .62244 .78426 .63380 .77110 .64607 .76260 .43 15 .60606 .70938 .62260 .78806 .63388 .77100 .64607 .76202 .43 16 .60680 .709335 .62220 .78806 .63308 .77100 .64607											
1	,	SINE	Cosine	SINE	Cosine	SINE	COSINE	SINE	Cosine	1	
1	0	.60182	.70864	.61566	.788or	.62032	.77715	.64270	76604	60	
2											
	2							.64323	.76567	58	
5											
6 6.0321 .90788 6.1704 .78044 .03068 .77005 .64412 .76402 54 7 60344 .90731 .61720 .78676 .63000 .77586 .64435 .76455 52 9 .60300 .79088 .61795 .78648 .63113 .77586 .64457 .76455 52 10 .60414 .79688 .61795 .78622 .03135 .77531 .64591 .76436 51 11 .60437 .70671 .61818 .78604 .63180 .77513 .64591 .76475 51 12 .6040 .79635 .61864 .78586 .63225 .77470 .61566 .76364 .73894 .61944 .60500 .70361 47 15 .60520 .79600 .61909 .78514 .63234 .77476 .61560 .70363 .41 16 .60535 .79536 .60195 .78146 .63316 .74421 .60622 <td>4</td> <td></td> <td></td> <td></td> <td></td> <td></td> <td></td> <td></td> <td></td> <td></td>	4										
7	5								70511		
8 60367 790723 61749 786458 6.3113 7.7568 6.4457 7.6455 52 9 60300 790706 61772 7.78640 6.3135 7.7550 6.4470 7.6417 50 11 60417 79671 6.1818 7.8664 6.3180 7.7513 6.4501 7.6417 50 11 60437 70671 6.1818 7.8664 6.3180 7.7513 6.4501 7.6417 50 11 60437 70671 6.1818 7.8664 6.3320 7.7444 6.4546 7.6388 40 12 60460 7.9653 6.1847 7.8550 6.3225 7.7476 6.4568 7.6367 47 14 60560 7.9618 6.1887 7.8550 6.3225 7.7476 6.4568 7.6367 47 15 60520 7.9600 6.1090 7.8532 6.3271 7.7430 6.4612 7.6323 45 16 60553 7.9583 6.1032 7.8514 6.3038 7.7342 6.4657 7.6284 44 17 60376 7.9505 6.1035 7.8476 6.3316 7.7402 6.4657 7.6284 44 18 60500 7.9512 6.02024 7.8406 6.3316 7.7306 6.4701 7.6248 41 19 6.6687 7.9404 6.2466 7.8424 6.3383 7.7384 6.4702 7.6220 40 21 6.668 7.9404 6.2046 7.8424 6.3383 7.7387 6.4764 7.6210 38 22 6.6668 7.9447 6.2060 7.8405 6.3428 7.7310 6.4768 7.6120 38 23 6.6714 7.9459 6.2020 7.8436 6.3418 7.7292 6.4700 7.6173 37 24 6.0738 7.9441 6.2115 7.856 6.3483 7.7237 6.4766 7.6210 38 25 6.6761 7.9424 6.2138 7.8315 6.3406 7.7235 6.4834 7.0133 35 26 6.0784 7.9450 6.2000 7.8437 6.3438 7.7230 6.4766 7.6210 38 27 6.0867 7.9388 6.2183 7.8315 6.3406 7.7235 6.4834 7.0135 35 28 6.0833 7.9071 6.2206 7.8276 6.3540 7.7235 6.4836 7.6003 31 29 6.0853 7.9333 6.2221 7.8261 6.3608 7.7125 6.4838 7.7013 6.4876 7.6007 33 20 6.0869 7.9338 6.2218 7.8261 6.3653 7.7190 6.4965 7.6003 31 20 6.0856 7.9335 6.2220 7.8275 6.3653 7.7190 6.4965 7.6003 31 20 6.0853 7.9335 6.2220 7.8265 6.3653 7.7144 6.4967 7.6022 20 21 6.0668 7.9046 6.2246 7.8266 6.3658 7.77070 6.3658 7.7014	7								.76473		
9 .60390 .70706 .61772 .78640 .63135 .77553 .64470 .76436 51 10 .60414 .79688 .61795 .78622 .63158 .77553 .64501 .76417 50 11 .60437 .70671 .61818 .78604 .63180 .77513 .64524 .76388 49 12 .60460 .79653 .61841 .78586 .63203 .77404 .64546 .76388 49 13 .60483 .70635 .61864 .78508 .63225 .77404 .64546 .76388 49 14 .60506 .79618 .61887 .78550 .63248 .77458 .64500 .76342 40 15 .60520 .79600 .61909 .78532 .63211 .77476 .64568 .76361 47 16 .60553 .70583 .61932 .78514 .63203 .77421 .64635 .76304 44 17 .60576 .70565 .61955 .78496 .63316 .77402 .64657 .76286 43 18 .60599 .70547 .61955 .78496 .63316 .77402 .64657 .76286 43 10 .60562 .79630 .6001 .78460 .63361 .77306 .64701 .76248 41 10 .60568 .70494 .62046 .78424 .63406 .77329 .64746 .76210 39 22 .60601 .704177 .62009 .78485 .63488 .77310 .64768 .76102 39 23 .60714 .70459 .62004 .78425 .63428 .77310 .64768 .76102 39 24 .60761 .709424 .62165 .78333 .63518 .77292 .64909 .76173 37 25 .60761 .709424 .62165 .78333 .63518 .777255 .64834 .70133 .32 25 .60761 .709424 .62165 .78333 .63518 .777255 .64843 .70133 .32 26 .60853 .70353 .62200 .78387 .63406 .771218 .64858 .70116 34 27 .60807 .70388 .62160 .78333 .63518 .777218 .64878 .76079 .33 28 .60830 .70351 .62206 .78207 .63563 .77109 .64091 .76028 32 29 .60853 .70353 .62200 .78207 .63563 .77109 .64091 .76028 32 20 .60853 .70353 .62200 .78207 .63563 .77109 .64091 .76028 32 20 .60853 .70353 .62200 .78207 .63563 .77109 .64091 .76028 32 20 .60853 .70353 .62200 .78207 .63563 .77109 .64091 .76028 32 21 .60668 .70338 .62264 .78207 .63688 .77162 .64045 .76010 33 28 .60830 .70353 .62200 .78207 .63568 .77181 .64069 .76020 32 29 .60853 .70353 .62200 .78207 .63608 .77100 .64069 .76020 32 20 .60853 .70353 .62200 .78207 .63608 .77100 .64091 .76028 32 20 .60853 .70353 .62200 .78207 .63608 .77100 .64095 .76020 32 21 .60668 .70888 .62183 .78315 .63608 .77100 .64095 .76020 32 22 .6001 .7006 .60000000000000000000000000000000	8				.78658						
11			.79706	.61772	.78640	.63135	.77550			51	
12	10	.60414	.79688	11	.78622	.63158	·77531	.64501	.76417	50	
13	11				.78604						
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16 .60553 .79883 .61932 .78814 .63203 .77421 .64655 .76364 41 18 .60599 .79547 .61978 .78478 .63316 .77422 .64657 .76267 42 10 .60522 .79530 .6201 .73460 .63316 .77360 .64761 .76267 42 20 .60645 .79512 .62024 .78442 .63383 .77340 .64766 .76229 40 21 .66668 .79494 .62046 .78424 .63406 .77329 .64746 .76229 40 22 .66061 .79479 .62092 .78387 .63416 .77329 .64766 .76270 .76360 .63473 .77320 .64768 .76173 37 23 .60714 .70459 .62002 .78386 .63437 .77273 .6482 .76173 37 25 .60761 .79424 .62138 .78315 .63468 .7723 <td></td> <td>62520</td> <td></td> <td></td> <td></td> <td></td> <td></td> <td></td> <td></td> <td></td>		62520									
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21	19				.78460				.76248		
22	20	.60645	.79512	.62024		11	•77347	.64723	.76229	40	
23	21									39	
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26											
27											
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32	-	1	1	11)	11 -				1 -	
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31 .60968 .79264					78206		-77107				
55 .60901 .70247 .62305 .78170 .63720 .77070 .65055 .75046 22 36 .61015 .79229 .62388 .78152 .63742 .77051 .65077 .75927 24 37 .61038 .79211 .62411 .78134 .63765 .77031 .65107 .75908 23 38 .61061 .79193 .62433 .7816 .63787 .77014 .65122 .75880 22 40 .61107 .79158 .62479 .78079 .63832 .76977 .65164 .778570 21 42 .61153 .79122 .62524 .78041 .63872 .76949 .65210 .75813 18 43 .61176 .79105 .62547 .78025 .63899 .76921 .65232 .757813 18 43 .61176 .79055 .62547 .78025 .63899 .76020 .65252 .75783 18 45					.78188		.77088			26	
36 .61015 .79229 .62388 .78512 .63742 .77051 .05977 .75908 23 37 .61038 .79211 .62411 .78134 .63765 .77933 .65100 .75908 23 38 .61061 .79193 .62433 .78116 .63787 .77014 .65122 .75890 23 40 .61107 .79158 .62479 .78079 .63810 .76996 .65144 .75870 21 41 .61130 .79140 .62502 .78061 .63844 .76950 .65188 .75831 12 42 .61133 .79105 .62524 .78043 .63877 .76940 .65210 .75831 18 43 .61176 .79105 .62527 .78075 .63892 .76930 .65232 .757794 17 45 .61222 .79069 .62502 .77088 .63044 .76847 .65240 .757751 17 44					.78170					25	
38	36		.79229		.78152						
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44 .61199 .79087 .62570 .78007 .63022 .76003 .65254 .75775 .6546 .61222 .79069 .62502 .77088 .63044 .76884 .65276 .75755 .15 .4646 .61245 .79051 .62615 .77070 .63066 .76840 .65298 .75738 .14 .47 .61268 .79033 .62638 .77952 .63080 .76847 .65320 .75710 .12 .48 .61291 .79016 .62660 .77034 .64011 .76828 .65342 .75700 .12 .49 .61314 .78008 .62683 .77016 .64033 .76810 .65344 .75608 .11 .50 .61337 .78080 .62706 .77897 .64050 .76701 .65366 .75561 .10 .51360 .78062 .62728 .77879 .64078 .76701 .65380 .75642 .9 .52 .61383 .78044 .62751 .77861 .64100 .76754 .65430 .75623 .8 .53 .61460 .78036 .62706 .77824 .64145 .76737 .65452 .75604 .75585 .55 .61451 .78801 .6280 .77882 .64150 .76050 .64078 .75656 .55 .61451 .78801 .6280 .77806 .64160 .76054 .65430 .75565 .55 .61451 .78801 .6280 .77806 .64160 .76608 .05406 .75566 .55 .61474 .78853 .62864 .77706 .64100 .706734 .65470 .75585 .6556 .61474 .78853 .62864 .77706 .64100 .706734 .65470 .75585 .6556 .61474 .78853 .62864 .77706 .64100 .70608 .05406 .75566 .55 .61474 .78853 .62864 .77706 .64100 .70608 .05406 .75566 .55 .61474 .78853 .62864 .77706 .64212 .706061 .65540 .75556 .55 .61451 .78830 .62000 .77835 .64450 .70602 .05518 .75528 .35 .61450 .78837 .62867 .77773 .64256 .76603 .05560 .75550 .55 .61520 .78837 .62867 .77775 .64256 .76623 .65552 .75580 .20804 .77706 .64212 .706061 .65540 .75550 .55 .61520 .78837 .62864 .77706 .64212 .706061 .65540 .75550 .55 .61520 .78837 .62867 .77775 .64226 .76603 .65560 .75550 .55 .61520 .78837 .62867 .77775 .64226 .76604 .65552 .75580 .20804 .77706 .64212 .706061 .65540 .75550 .55 .60666 .78801 .62000 .77733 .64256 .76603 .65606 .75570 .75570 .60000 .62000 .77733 .64256 .76603 .65606 .75570 .75570 .600000 .62000 .77733 .64256 .76603 .65606 .75570 .75570 .600000000000000000000000000000000000											
45 .61222 .79069				.62570	.78007		.76903	.65254	·75775	16	
47 .61268 .79033 .62638 .77952 .63989 .76847 .65320 .758710 13 48 .61291 .79016 .62660 .77934 .64011 .76828 .65342 .75700 13 49 .61314 .78908 .62683 .77916 .64033 .76810 .63384 .75700 50 .61337 .78980 .62706 .77897 .64056 .76701 .65386 .75680 51 .61360 .78962 .62728 .77879 .64056 .76701 .65386 .75661 52 .61383 .78044 .62751 .77861 .64100 .76754 .65430 .75623 53 .61406 .78026 .62774 .77843 .64123 .76735 .65452 .75604 54 .61429 .78008 .62706 .77824 .64145 .76717 .65474 .75585 55 .61451 .78891 .62819 .77806 .64107 .76698 .05400 .75546 55 .61451 .78873 .62842 .77788 .64100 .76679 .65518 .75547 56 .61474 .78873 .62842 .77788 .64100 .76679 .65518 .75547 57 .61497 .78855 .62804 .77750 .64214 .76601 .65540 .75528 58 .61520 .78837 .62887 .77751 .64234 .76642 .65562 .75509 2 .75807 .61566 .78801 .62909 .77733 .64256 .76623 .65584 .75590 2 .75808 .61566 .78801 .62902 .77715 .64234 .76642 .65562 .75509 2 .75808 .61566 .78801 .62902 .77715 .64236 .76642 .65560 .75471 0	45	.61222	.79069	.62592	.77988						
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40 .61314 .78098 .62683 .77016 .64033 .76810 .65364 .75680 I10 50 .61337 .78980 .62706 .77897 .64056 .76791 .65386 .75661 I10 51 .61360 .78962 .62728 .77879 .64078 .76772 .65408 .75642 9 52 .61383 .78044 .62751 .77861 .64100 .76754 .65430 .75024 53 .61406 .78026 .62774 .77843 .64123 .76735 .65452 .75604 7 54 .61429 .78008 .62796 .77824 .64145 .76717 .65474 .75885 6 55 .61451 .78891 .62819 .77806 .64107 .76608 .05406 .75566 5 56 .61474 .78873 .62842 .77788 .64190 .70679 .05518 .75526 3 58 .61520 .78837 .62864 .77769 .64212 .76061 .65540 .75526 3 58 .61520 .78837 .62804 .77769 .64212 .76061 .65540 .75550 5 59 .61543 .78810 .62000 .77733 .64256 .76623 .65502 .75500 1 60 .61566 .78801 .62000 .77715 .64274 .76624 .55502 .75590 1	47		.79033								
50 .61337 .78980 .62706 .77897 .64956 .76701 .65386 .75661 10 51 .61360 .78962 .62728 .77879 .64978 .76772 .65498 .75642 9 52 .61383 .789044 .62751 .77851 .64100 .76734 .6439 .75023 8 53 .61406 .78026 .62774 .77843 .64123 .76735 .65452 .75604 7 54 .61429 .78806 .62706 .77824 .64145 .76171 .65474 .75856 5 55 .61451 .78891 .62812 .77788 .64109 .76679 .65540 .75566 5 57 .61497 .78855 .62842 .77769 .64212 .76661 .65540 .75528 3 58 .61520 .78837 .62887 .77751 .64236 .76624 .65562 .75590 2 59 .6			78008						-7568c		
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53										8	
55		.61406	.78926	.62774	.77843		.76735	.65452	.75604	7	
56 6.1474 .78873 .62842 .77788 .64190 .76679 .65518 .75547 4 57 .61497 .78855 .62864 .77769 .64212 .76661 .65540 .75528 3 58 .61520 .78837 .62887 .77751 .64234 .76642 .65562 .75599 2 59 .61543 .78810 .62909 .777733 .64256 .76623 .65584 .75490 1 6 .61566 .78801 .62932 .77715 .64279 .76604 .5606 .75471 0 7 .6080E .8080E .8080E .8080E .8080E .8080E .75471 0	54	.61429	.78008	.62796	.77824				.75585		
57	55									5	
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59 .61543 .758510 .62909 .77733 .64256 .76023 .65584 .75490 .76060 .61566 .75801 .62932 .77715 .64279 .76604 .65606 .75471 0	57		78827					.65562		2	
60 .61566 .78801 .62932 .77715 .64279 .76604 .65606 .75471 0											
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		Cocnia	Cnip	Cogny	Cryp	Cocnya	SINE	Cosine	SINE		
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,	SINE	Cosine		Cosine	11	Cosine		Cosine	
0	.65606	.75471	.66913	.74314	.68200	.73135	.69466	-71934	60
I 2	.65628	.75452	.66935	.74295	.68221	.73116	.69487	.71914	59
3	.65672	·75433	.66978	.74270	.68264	.73076	.69529	.71894	58 57
4	.65694	.75395	.66000	.74227	.68235	.73056	.69549	.71853	56
	.65716	.75375	.67021	-74217	.68306	.73036	.69570	.71833	55
5 6	.65738	.75356	.670.43	.74198	.68327	.73016	.69591	.71813	54
7 8	.65759	-75337	.67064	.74178	.68349	.72996	.69612	.71792	53
	.65781	.75318	.67086	-74159	.68370	.72976	.69633	.71772	52
9 10	.65803	.75299	.67107	·74139	.68391	.72957	.69654	.71752	51
	.65825	.75280	.57129		11	.72937	69675	.71732	50
11	.65847	.75261	.67151	.74100	.68434	.72917	.69696	.71711	49
13	.65801	.75222	.67194	.74061	.68476	.72377	.69737	.71691	48
14	.65913	.75203	.67215	.74041	.68497	.72857	.69758	.71650	46
15	.65935	.75184	.67237	.74022	.68518	.72837	.69779	.71630	45
16	.65956	.75165	.67258	.;4002	.68539	.72817	.69800	.71610	44
17	.65978	75146	.67280	.73983	.68561	.72797	.69821	.71590	43
18	.66000	.75126	.67301	.73963	.68582	-72777	.69842	.71569	42
20	.66022	.75107	.67323	-73944	.68603	-72757	.69862	-71549	41
	.66066	.75088	.67344	-73924	.68624	-72737		.71529	40
21	.66088	.75069	.67366	.73904 .73885	.68645	.72717	.69904	.71508	39
23	.66100	.75050 .75030	.67400	.73865	.68688	.72697	.69925	.71488 .71468	38
24	.66131	.75011	.67430	.73846	.68700	.72657	.69946	.71447	37 36
25	.66153	.74992	.67452	.73826	.68730	.72637	.69987	-71427	35
26	.66175	.74973	.67473	.73806	.68751	.72617	.70008	.71407	34
27	.66197	·74953	.67495	.73787	.68772	.72597	.70029	.71386	33
28	.66218	•74934	.67516	.73767	.68793	.72577	.70049	.71366	32
29	.66240	·74915	.67538	·73747	.68814	·72557	.70070	.71345	31
30	.66284	.74896	.67559	.73728	.68835	•72537	.70091	.71325	30
31 32	.66306	.74876 .74857	.67580 .67602	.73708 .73688	.68878	.72517 .72497	.70112	.71305 .71284	29 28
3 3	.66327	.74838	.67623	.73669	.68899	.72477	.70153	.71264	27
34	.66349	.74818	.67645	.73649	.68020	.72457	.70174	.71243	26
35	.66371	-74799	.67666	.73629	.68941	.72437	.70195	.71223	25
36	.66393	-74780	.67688	.73610	.68962	.72417	.70215	.71203	24
37	.66414	-74760	.67709	.73590	.68983	-72397	.70236	.71182	23
38 39	.66436 .66458	-74741	.67730	.73570	.6900.4	-72377	.70257	.71162	22
40	.66480	.74722 .74703	.67752	.73551 .73531	.69025	·72357 ·72337	.70277	.71141	2 I 20
41	.66501	.74683	.67795	.73511	.69067	.72317	.70319	.71100	19
42	.66523	.74664	.67816	.73491	.69088	.72207	.70319	.71100	18
43	.66545	.74644	.67837	.73472	.69100	.72277	.70360	.71050	17
44	.66566	.7.4625	.67859	.73452	.69130	.72257	.70381	.71039	16
45	.66588	.74606	.67880	.73432	.69151	.72236	.70401	.71019	15
46	.66610	.74586	.67901	.73413	.69172	.72216	.70422	.70998	14
47 48	.66632 .66653	.74567	.67923	.73393	.69193	.72196	.70443	.70978	13
49	.66675	.74548 .74528	.67944 .67965	·73373 ·73353	.69214	.72176 .72156	.70463	.70957	12 11
50	.66697	.74500	.67987	.73333	.69256	.72136	.70505	.70016	10
51	.66718	.74489	.68008	.73314	.60277	.72116	.70525	.70806	
52	.66740	.74470	.68020	.73294	.60208	.72095	.70546	.70875	8
5 3	.66762	·74451	.68051	.73274	.69319	.72075	.70567	.70855	7
54	.66783	.74431	.68072	.73254	.69340	-72055	.70587	.70834	
55	.66805	.74412	.68093	.73234	.69361	.72035	.70608	.70813	5
56 57	.66827 .66848	.74392	.68115	.73215	.69382	.72015	.70628	.70793	4
58	.66870	·74373 ·74353	.68136	.73195	.69403 .69424	.71995 .71974	.70670	.70772	3
59	.66891	·74333	.68179	.73175	.69445	.71974	.70690	.70731	I
60	.66913	.74314	.68200	.73135	.69466	.71934	.70711	.70711	0
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0	I	Infinite.	1.0001	57.200	1.0006	28.654	1.0014	19.107	60
. 1	I	3437.70	1.0001	56.359	1.0006	28.417	1.0014	19.002	59
2	1	1718.90	1.0002	55.450	1.0006	28.184	1.0014	18.897	58
3	I	1145.90	1.0002	54.570	1.0006	27.955	1.0014	18.794	57
4	I	859.44	1.0002	53.718	1.0006	27.730	1.0014	18.692	56
5 6	I	687.55 572.96	1.0002	52.891	1.0007	27.508	1.0014	18.491	55 54
7	ī	491.11	1.0002	51.313	1.0007	27.075	1.0015	18.393	53
8	1	429.72	1.0002	50.558	1.0007	26.864	1.0015	18.295	52
9	I	381.97	1.0002	49.826	1.0007	26.655	1.0015	18.198	51
10	1	343.77	1.0002	49.114	1.0007	26.450	1.0015	18.103	50
11	1	312.52	1.0002	48.422	1.0007	26.249	1.0015	18.008	49
12	I	286.48	1.0002	47.750	1.0007	26.050	1.0016	17.914	48
14	I	264.44	1.0002	47.096 46.460	1.0007	25.661	1.0016	17.730	47 46
15	ī	22 2.18	1.0002	45.840	1.0008	25.471	1.0016	17.639	45
16	ı	214.86	1.0002	45-237	1.0008	25.284	1.0016	17.549	44
17	I	202.22	1.0002	44.650	1.0008	25.100	1.0016	17.460	43
18	1	190.99	1.0002	44.077	1.0008	24.918	1.0017	17.372	42
19	1	180.73	1.0003	43.520	1.0008	24.739	1.0017	17.285	41
. 20	I	171.89	1.0003	42.076	1.0008	24.562	1.0017	17.198	40
2 I 22	I	163.70 156.26	1.0003	42.445 41.928	1.0008	24.358	1.0017	17.113	39 38
23	ī	149.47	1.0003	41.423	1 0000	24.047	1.0017	16.944	37
24	I	143.24	1.0003	40.930	1.0000	23.880	1.0018	16.861	36
25	1	137.51	1.0003	40.448	1.0000	23.716	1.0018	16.779	35
26	I	132.22	1.0003	39.978	1.0000	23.553	1.0018	16.698	34
27 28	I	127.32	1.0003	39.518	1.0000	23.393	1.0018	16.617	33
20 20	I	122.78	1.0003	39.069 38.631	1.0000	23.235	1.0018	16.538	32 31
30	I	114.59	1.0003	38.201	1.0009	22.025	1.0010	16.380	30
31	1	110.90	1.0003	37.782	1.0010	22.774	1.0019	16.303	29
32	1	107.43	1.0003	37.371	1.0010	22.624	1.0019	16.226	28
33	I	104.17	1.0004	36.969	1.0010	22.476	1.0019	16.150	27
34	I	101.11	1.0004	36.576 36.101	1.0010	22.330	1.0019	16.000	26
35 36	I	95:223 95:405	1.0004	35.814	1.0010	22.044	1.0019	15.026	25 24
37	ī	92.914	1.0004	35.445	1.0010	21.904	1.0020	15.853	23
38	1.0001	92.469	1.0004	35.084	1.0010	21.765	1.0020	15.780	22
39	1.0001	88.149	1.0004	34.729	1.0011	21.629	1.0020	15.708	2 I
40	1.0001	85.946	1.0004	34.382	1.0011	21.494	1.0020	15.637	20
41	1.0001	83.849	1.0004	34.042	1.0011	21.360	1.0021	15.566	10
42 43	1.000.1	81.853	1.0004	33.708	1.0011	21.228	1.0021	15.496	18 17
43	1.0001	78.133	1.0004	33.381 33.060	1.0011	20.970	1.0021	15.358	16
45	1.0001	76.396	1.0005	32.745	1.0011	20.843	1.0021	15.290	15
46	1.0001	74.736	1.0005	32.437	1.0012	20.717	1.0022	15.222	14
47	1.0001	73.146	1.0005	. 32-134	1.0012	20.593	1.0022	15.155	13
48	1.0001	71.622	1.0005	31.836	1.0012	20.471	1.0022	15.089	12
49 50	1.0001	71.160 68.757	1.0005	31.544	1.0012	20.350	1.0022	15.023	11
51	1.0001	67.400	1.0005	30.076	1.0012	20.112	1.0023	14.803	
52	1.0001	66.113	1.0005	30.699	1.0012	19.995	1.0023	14.829	8
53	1.0001	64.866	1.0005	30.428	1.0013	19.880	1.0023	14.765	6
54	1.0001	63.664	1.0005	30.161	1.0013	19.766	1.0023	14.702	
55 56	1.0001	62 507	1.0005	29.899	1.0013	19.653	1.0023	14.640	5
57	1.0001	61.391	1.0006	29.388	1.0013	19.541	1.0024	14.578	4
58	1.0001	59.274	1.0006	29.139	1.0013	19.322	1.0024	14.456	2
59	1.0001	58.270	1.0006	28.894	1.0013	19.214	1.0024	14.395	1
60			1.0006	28.654	1.0014	19.107	1.0024	14.335	•
•	, CO-SEC SEC		Co-sec.	SEC.	Co-sec.	SEC.	Co-sec.	SEC.	
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,	SEC.	Co-sec.	SEC.	Co-sec.	SEC.	Co-sec	SEC.	Co-sec.	,
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0	1.0024	14.335	1.0038	11.474	1.0055	9.5668	1.0075	8.2055	60 59
1 2	1.0025	14.276	1.0030	11.398	1.0056	9.5141	1.0076	8.1668	58
3	1.0025	14.150	1.0030	11.360	1.0056	9.4880	1.0076	8.1476	57
4	1.0025	14.101	1.0030	11.323	1.0056	9.4620	1.0076	8.1285	56
5 6	1.0025	14.043	1.0039	11.286	1.0057	9.4362	1.0077	8.1004	55
6	1.0026	13.986	1.0040	11.249	1.0057	9.4105	1.0077	8.0905	54
7 8	1.0026	13.930	1.0040	11.213	1.0057	9.3506	1.0078	8.0529	53 52
9	1.0026	13.818	1.0040	11.140	1.0058	9.3343	1.0078	8.0342	51
10	1.0026	13.763	1.0041	11.104	1.0058	9.3092	1.0079	8.0156	50
11	1.0027	13.708	1.0041	11.069	1.0058	9.2842	1.0079	7.9971	49
12	1.0027	13.654	1.0041	11.033	1.0059	9.2593	1.0079	7.9787	48
13	1.0027	13.600	1.0041	10.988	1.0059	9.2346	1.0080	7.9604	47
14	1.0027	13.547	1.0042	10.963	1.0050	0.1855	1.0080	7.9421	46 45
15 16	1.0027	13.494 13.441	I.0042 I.0042	10.804	1.0060	0.1612	1.0081	7.3250	45
17	1.0028	13.380	1.0043	10.860	1.0060	9.1370	1.0081	7.8870	43
18	1.0028	13.337	1.0043	10.826	1.0061	9.1129	1.0082	7.8700	42
19	1.0028	13.286	1.0043	10.792	1.0061	0.0800	1.0082	7.8522	41
20	1.0029	13.235	1.0043	10.758	1.0061	9.0651	1.0082	7.8344	40
21	1.0029	13.184	1.0044	10.725	1.0062	9.0414	1.0083	7.8168	39
22	1.0029	13.134	1.0044	10.602	1 0062	9.0179	1.0083	7.7992	38
23	1.0029	13.084	1.0044	10.659	1.0062	8.9944	1.0084	7.7817	37
24 25	1.0029	13.034	1.0044	10.626	1.0063	8.9479	1.0084	7.7469	36 35
26	1.0030	12.935	1.0045	10.561	1.0063	8.0248	1.0085	7.7296	34
27	1.0030	12.888	1.0045	10.529	1.0064	8.0018	1.0085	7.7124	33
28	1.0030	12.840	1.0046	10.497	1.0064	8.8790	1.0085	7.6953	32
29	1.0031	12.793	1.0046	1.0046 10.465		8.8563	1.0086	7.6783	3 I
30	1.0031	1.0031 12.745 1.0046 10.433		10.433	1.0065	8.8337	1.0086	7.6613	30
31	1.0031	12.698	1.0046	10.402	1.0065	8.8112	1.0087	7.6444	20
32	1.0031	12.652	1.0047	10.371	1.0065	8.7888 8.7665	1.0087	7.6276	28
33	1.0032	12.606	1.0047	10.340	1.0066	8.7444	1.0088	7.5942	27 26
34 35	1.0032	12.514	1.0048	10.278	1.0066	8.7223	1.0088	7.5776	25
35	1.0032	12.460	1.0048	10.248	1.0067	8.7004	1.0080	7.5611	24
37	1.0032	12.424	1.0048	10.217	1.0067	8.6786	1.0089	7.5446	23
38	1.0033	12.379	1.0048	10.187	1.0067	8.6569	1.0089	7.5282	22
39	1.0033	12.335	1.0049	10.157	1.0068	8.6353	1.0000	7.5119	21
40	1.0033	12.291	1.0049	10.127	1.0068	8.6138	1,0000	7.4957	20
41	1.0033	12.248	1.0049	10.098	1.0068	8.5924	1.0000	7.4795	18
42 43	1.0034	12.204 12.161	1.0050	10.068	1.0069	8.5711 8.5499	1.0001	7.4634 7.4474	17
44	1.0034	12.118	1.0050	10.010	1.0060	8.5289	1.0002	7.4315	16
45	1.0034	12.076	1.0050	9.9812	1.0070	8.5079	1.0002	7.4156	15
46	1.0035	12.034	1.0051	9.9525	1.0070	8.4871	1.0092	7.3998	14
47	1.0035	11.992	1.0051	9.9239	1.0070	8.4663	1.0093	7.3840	13
48	1.0035	11.950	1.0051	9.8955	1.0071	8.4457	1.0093	7.3683	12
49 50	1.0035	11.909	1.0052	9.8672 9.8391	1.0071	8.4251 8.4046	1.0094	7.3527 7.3372	11
51	1.0036	11.828	1.0052	0.8112	1.0072	8.3843	1.0004	7.3217	
52	1.0036	11.787	1.0053	9.7834	1.0072	8.3640	1.0005	7.3063	9 8
5 3	1.0036	11.747	1.0053	9.7558	1.0073	8.3439	1.0095	7.2909	7 6
54	1.0037	11.707	1.0053	9.7283	1.0073	8.3238	1.0096	7.2757	
55	1.0037	11.668	1.0053	9.7010	1.0073	8.3039	1.0096	7.2604	5
56	1.0037	11.628	1.0054	9.6739	1.0074	8.2840	1.0097	7.2453	4
57 58	1.0037	11.589	1.0054	9.6469 9.6200	1.0074	8.2642 8.2446	1.0097	7.2302	3
59	1.0038	11.550	1.0054	9.5933	1.0074	8.2250	1.0007	7.2002	1
60	1.0038	11.474	1.0055	9.5668	1.0075	8.2055	1.0098	7.1853	0
-	Co-sec.	SEC.	Cospe	Spa	Coana	Cno	Coarc	Spo	-
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0	1.0008	7.1853	1.0125	6.3924	1.0154	5.7588	1.0187	5.2408	6
1	1.0000	7.1704	1.0125	6.3807	1.0155	5.7493	1.0188	5.2330	5
2	1.0000	7.1557	1.0125	6.3690	1.0155	5.7398	1.0188	5.2252	5
3	1.0009	7.1409	1.0126	6.3574	1.0156	5.7304	1.0189	5-2174	5
4	1.0100	7.1263	1.0126	6.3458	1.0156	5.7210	1.0189	5.2097	50
5 6	1.0100	7.1117	1.0127	6.3343	1.0157	5.7117	1.0190	5.2019	5
6	1.0101	7.0972	1.0127	6.3228	1.0157	5.7023	1.0191	5.1942	54
7	1.0101	7.0827	1.0128	6.3113	1.0158	5.6930	1.0191	5.1865	53
	1.0102	7.0683	1.0128	6.2999	1.0158	5.6838	1.0192	5.1788	5
9	1.0102	7.0539	1.0129	6.2885	1.0159	5.6745	1.0192	5.1712	5
10	1.0102	7.0396	1.0129	6.2772	1.0159	5.6653	1.0193	5.1636	59
11	1.0103	7.0254	1.0130	6.2659	1.0160	5.6561	1.0193	5.1560	49
12	1.0103	7.0112	1.0130	6.2546	1.0160	5.6470	1.0194	5.1484	48
13	1.0104	6.9971	1.0131	6.2434	1.0161	5.6379	1.0195	5.1409	47
14	1.0104	6.9830	1.0131	6.2322	1.0162	5.6288	1.0195	5.1333	46
15	1.0104	6.9690	1.0132	6.2211	1.0162	5.6197	1.0196	5.1258	45
16	1.0105	6.9550	1.0132	6.2100	1.0163	5.6107	1.0196	5.1183	44
17 18	1.0105	6.9411	1.0133	6.1990 6.1880	1.0163	5.6017	1.0197	5.1100	43
10	1.0106	6.9273	1.0133		1.0164	5.5928	1.0198	5.1034	42
20	1.0107	6.8008	1.0134	6.1770 6.1661	1.0165	5.5749	1.0198	5.0886	41
				1			11		1 .
21	1.0107	6.8861	1.0135	6.1552	1.0165	5.5660	1.0199	5.0812	39
22	1.0107	6.8725	1.0135	6.1443	1.0166	5.5572	1.0200	5.0739	38
23	1.0108	6.8589	1.0136	6.1335	1.0166	5.5484	1.0201	5.0666	37
24	1.0108	6.8454 6.8320	1.0136	6.1227	1.0167	5.53)6	1.0201	5.0593	36
25 26	1.0100	6.8185	1.0136	6.1013	1.0167	5.5308	1.0202	5.0520	35
27	1.0110	6.8052	1.0137	6.0006	1.0160	5.5134	1.0203	5.0447	33
28	1.0110	6.7919	1.0138	6.0800	1.0169	5.5047	1.0203	5.0302	32
29	1.0111	6.7787	1.0138	6.0694	1.0170	5.4960	1.0204	5.0230	31
30	1.0111	6.7655	1.0130	6.0588	1.0170	5.4874	1.0205	5.0158	30
31	1.0111	6.7523	1.0139	6.0483	1.0171	5.4788	1.0205	5.0087	29
32	1.0111	6.7392	1.0130	6.0379	1.0171	5.4702	1.0205	5.0015	28
33	1.0112	6.7262	1.0140	6.0274	1.0172	5.4617	1.0207	4.9944	27
34	1.0113	6.7132	1.0141	6.0170	1.0172	5.4532	1.0207	4.9873	26
35	1.0113	6.7003	1.0141	6.0066	1.0173	5.4447	1.0208	4.9802	25
36	1.0114	6.6874	1.0142	5.9963	1.0174	5.4362	1.0208	4.9732	24
37	1.0114	6.6745	1.0142	5.9860	1.0174	5.4278	1.0200	4.9661	23
38	1.0115	6.6617	1.0143	5.9758	1.0175	5.4194	1.0210	4.9591	22
39	1.0115	6.6490	1.0143	5.9655	1.0175	5.4110	1.0210	4.9521	21
40	1.0115	6.6363	1.0144	5.9554	1.0176	5.4026	1.0211	4.9452	20
41	1.0116	6.6237	1.0144	5.9452	1.0176	5-3943	1.0211	4.9382	19
42	1.0116	6.6111	1.0145	5.9351	1.0177	5.3860	1.0212	4.9313	18
43	1.0117	6.5985	1.0145	5.9250	1.0177	5.3777	1.0213	4.9243	17
44	1.0117	6.5860	1.0146	5.9150	1.0178	5.3695	1.0213	4.9175	16
45	1.0118	6.5736	1.0146	5.0049	1.0179	5.3612	1.0214	4.9106	15
46	1.0118	6.5612	1.0147	5.8950	1.0179	5.3530	1.0215	4.9037	14
47	1.0119	6.5488	1.0147	5.8850	1.0180	5 3449	1.0215	4.8969	13
48	1.0119	6.5365	1.0148	5.8751	1.0180	5.3367	1.0216	4.8901	12
49	1.0110	6.5243	1.0148	5.8652	1.0181	5.3286	1.0216	4.8833	11
50	1.0120	6.5121	1.0149	5.8554	1.0181	5.3205	1.0217	4.8765	10
51	1.0120	6.4000	1.0150	5.8456	1.0182	5.3124	1.0218	4.8697	8
52	1.0121	6.4878	1.0150	5.8358	1.0182	5.3044	1.0218	4.8630	
53	1.0121	6.4757	1.0151	5.8261	1.0183	5.2063	1.0219	4.8563	7 6
54	1.0122	6.4637	1.0151	5.8163	1.0184	5.2883	1.0220	4.8496	
55 56	1.0122	6.4517	1.0152	5.8067	1.0184	5.2803	1.0220	4.8429	5
57	1.0123	6.4398	1.0152	5.7970	1.0185	5.2724	1.0221	4.8296	4
58	1.0123	6.4160	1.0153	5.7874	1.0186	5.2566	1.0221	4.8220	3
59	1.0124	6.4042	1.0153	5.7683	1.0186	5.2487	1.0223	4.8163	ī
60	1.0125	6.3924	1.0154	5.7588	1.0187	5.2408	1.0223	4.8097	ō
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0	1.0223	4.8097	1.0263	4.4454	1.0306	4.1336	1.0353	3.8637	60
1	1.0224	4.8032	1.0264	4.4398	1.0307	4.1287	1.0353	3.8595	59
3	1.0225	4.7966 4.7901	1.0264	4.4342	1.0308	4.1239 4.1101	1.0354	3.8553	58 57
4	1.0226	4.7835	1.0266	4.4231	1.0300	4.1144	1.0356	3.8470	56
5	1.0226	4.7770	1.0266	4.4176	1.0310	4.1096	1.0357	3.8428	55
6	1.0227	4.7706	1.0267	4.4121	1.0311	4.1048	1.0358	3.8387	54
7 8	1.0228	4.7641	1.0268	4.4065	1.0311	4.100 I 4.00 5 3	1.0358	3.8346 3.8304	53 52
9	1.0220	4.7512	1.0269	4.3056	1.0313	4.0006	1.0300	3.8263	51
10	1.0230	4.7448	1.0270	4.3910	1.0314	4.0859	1.0361	3.8222	50
11	1.0230	4.7384	1.0271	4.3847	1.0314	4.0812	1.0362	3.8181	49
12	1.0231	4.7320	1.0271	4.3792	1.0315	4.0765	1.0362	3.8140	48
13 14	1.0232	4.7257 4.7193	1.0272	4.3738	1.0316	4.0718	1.0363	3.8100 3.8059	47 46
15	1.0233	4.7130	1.0273	4.3630	1.0317	4.0625	1.0365	3.8018	45
16	1.0234	4.7067	1.0274	4.3576	1.0318	4.0579	1.0366	3.7978	44
17	1.0234	4.7004	1.0275	4.3522	1.0319	4.0532	1.0367	3.7937	43
10	1.0235	4.6942	1.0276	4.3469	I.0320	4.0486	1.0367	3.7897 3.7857	42 41
20	1.0236	4.6817	1.0277	4.3362	1.0321	4.0394	1.0369	3.7816	40
2 I	1.0237	4.6754	1.0278	4.3300	1.0322	4.0348	1.0370	3.7776	39
22	1.0237	4.6692	1.0278	4.3256	1.0323	4.0302	1.0371	3.7736	38
23	1.0238	4.6631	1.0279	4.3203	1.0323	4.0256	1.0371	3.7697 3.7657	37 36
25	1.0239	4.6507	1.0280	4.3098	1.0325	4.0165	1.0373	3.7617	35
26	1.0240	4.6446	1.0281	4.3045	1.0326	4.0120	1.0374	3.7577	34
27	1.0241	4.6385	1.0282	4.2993	1.0327	4.0074	1.0375	3.7538	33
28 20	1.0241	4.6324	1.0283	4.2941	1.0327 1.0328	3.9984	1.0376	3.7498 3.7459	32 31
30	1.0242	4.6202	1.0284	4.2836	1.0320	3.9939	1.0377	3.7420	30
31	1.0243	4.6142	1.0285	4.2785	1.0330	3.9894	1.0378	3.7380	29
32	1.0244	4.6081	1.0285	4.2733	1.0330	3.9850	1.0379	3.7341	28
33	1.0245	4.6021 4.5961	1.0286	4.2681	1.0331	3.9805	1.0380	3.7302	27 26
34 35	1.0245	4.5901	1.0288	4.2630	1.0332	3.9760 3.9716	1.0381	3.7263 3.7224	25
36	1.0247	4.5841	1.0288	4.2527	1.0334	3.9672	1.0382	3.7186	24
37	1.0247	4.5782	1.0289	4.2476	1.0334	3.9627	1.0383	3.7147	23
38	1.0248	4.5722	1.0200	4.2425	1.0335	3.9583	1.0384	3.7108	22
39 40	1.0249	4.5663	1.0291	4.2375	1.0336	3.9539 3.9495	1.0385	3.7070 3.7031	2 I 20
41	1.0250	4.5545	1.0202	4.2273	1.0338	3.9451	1.0387	3.6993	19
42	1.0251	4.5486	1.0203	4.2223	1.0338	3.9408	1.0387	3.6955	18
43	1.0251	4.5428	1.0293	4.2173	1.0339	3.9364	1.0388	3.6917	17
44	1.0252	4.5369	1.0294	4.2122	1.0340	3.9320	1.0389	3.6878	16
45 46	1.0253	4.5311 4.5253	1.0295	4.2072	1.0341	3.9277 3.9234	1.0390	3.6840	14
47	1.0254	4.5195	1.0296	4.1072	1.0341	3.9199	1.0391	3.6765	13
48	1.0255	4.5137	1.0297	4.1923	1.0343	3.9147	1.0393	3.6727	I 2
49	1.0255	4.5079	1.0298	4.1873	1.0344	3.0104	1.0393	3.6689	ΙI
50	1.0256	4.5021	1.0200	4.1824	1.0345	3.9061	1.0394	3.6651	10
51 52	1.0257	4.4964 4.4997	1.0299	4.1774	1.0345	3.0018 3.8076	1.0305	3.6614 3.6576	8
53	1.0258	4.4850	1.0300	4.1725	1.0346	3.8033	1.0300	3.6530	7
54	1.0259	4.4793	1.0302	4.1627	1.0348	3.8000	1.0398	3.6502	7 6
55	1.0260	4.4736	1.0302	4.1578	1.0349	3.8848	1.0300	3.6464	5
56 57	1.0260	4.4679	1.0303	4.1529	1.0349	3.8805 3.8763	1.0399	3.6427	4
58 58	1.0261	4.4566	1.0304	4.1432	1.0350	3.8721	1.0400	3.6353	3 2
59	1.0262	4.4510	1.0305	4.1384	1.0352	3.8679	1.0402	3.6316	I
60	1.0263	4.4454	1.0306	4.1336	1.0353	3.8637	1.0403	3.6279	0
,	Co-sec.	SEC.	Co-sec.	SEC.	Co-sec.	SEC.	Co-sec.	SEC.	'

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′	SEC.	Co-sec.		Co-sec.		Co-sec.	Sec.	Co-sec.	•
0	1.0403	3.6270	1.0457	3.4203	1.0515	3.2361	1.0576	3.0715	60
1	1.0404	3.6243	1.0458	3.4170	1.0516	3.2332	1.0577	3.0690	59
2	1.0405	3.6206	1.0459	3.4138	1.0517	3.2303	1.0578	3.0664	58
3	1.0406	3.6169	1.0460	3.4106	1.0518	3.2274	1.0579	3.0638	57
4	1.0406	3.6133	1.0461	3.4073	1.0519	3.2245	1.0580	3.0612	56
5	1.0407	3.6096	1.0461	3.4041	1.0520	3.2216	1.0581	3.0586	55
6	1.0408	3.6060	1.0462	3.4000	1.0521	3.2188	1.0582	3.0561	54
7 8	1.0400	3.6024	1.0463	3.3977	1.0522	3.2159	1.0584	3.0535	53
9	1.0410	3.5987	1.0464	3.3945	1.0523	3.2131	1.0586	3.0509	52 51
10	1.0412	3.5915	1.0466	3.3881	1.0525	3.2074	1.0587	3.0458	50
11	1.0413	3.5879	1.0467	3.3849	1.0526	3.2045	1.0588	3.0433	49
12	1.0413	3.5843	1.0468	3.3817	1.0527	3.2017	1.0580	3.0433	48
13	1.0414	3.5807	1.0469	3.3785	1.0528	3.1989	1.0500	3.0382	47
14	1.0415	3.5772	1.0470	3.3754	1.0529	3.1960	1.0591	3.0357	46
15	1.0416	3.5736	1.0471	3.3722	1.0530	3.1932	1.0592	3.0331	45
16	1.0417	3.5700	1.0472	3.3600	1.0531	3.1904	1.0593	3.0306	44
17 18	1.0418	3.5665	1.0473	3.3659	1.0532	3.1876	1.0594	3.0281	43
	1.0419	3.5629	1.0474	3.3627	1.0533	3.1848	1:0595	3.0256	42
19	1.0420	3.5594	1.0475	3.3596	1.0534	3.1820	1.0596	3.0231	4E
20	1.0420	3.5559	1.0476	3.3505	1.0535	3.1792	1.0598	3.0206	40
2 I	1.0421	3.5523	1.0477	3.3534	1.0536	3.1764	1.0599	3.0181	39
22 23	1.0422	3.5488	1.0478	3.3502 3.3471	1.0537	3.1736	1.0600	3.0156	38
24	1.0423	3.5453 3.5418	1.0479	3.3440	1.0530	3.1681	1.0602	3.0106	37 36
25	1.0425	3.5383	1.0480	3.3409	1.0540	3.1653	1.0603	3.0081	35
26	1.0426	3.5348	1.0481	3.3378	1.0541	3.1625	1.0604	3.0056	34
27	1.0427	3.5313	1.0482	3.3347	1.0542	3.1598	1.0605	3.0031	33
28	1.0428	3.5279	1.0483	3.3316	1.0543	3.1570	1.0606	3.0007	32
29	1.0428	3.5244	1.0484	3.3286	1.0544	3.1543	1.0607	2.9982	31
30	1.0429	3.5209	1.0485	3.3255	1.0545	3.1515	1.0608	2.9957	30
31	1.0430	3.5175	1.0486	3.3224	1.0546	3.1488	1.0609	2.9933	20
32	1.0431	3.5140	1.0487	3.3194	1.0547	3.1461	1.0611	2.9908	28
33	1.0432	3.5106	1.0488	3.3163	1.0548	3.1433	1.0612	2.9884	27
34	1.0433	3.5072	1.0489	3.3133	1.0549	3.1406	1.0613	2.9859	26 25
35 36	1.0434	3.5037	1.0491	3.3072	1.0551	3.1352	1.0615	2.9810	24
37	1.0436	3.4969	1.0402	3.3042	1.0552	3.1325	1.0616	2.9786	23
38	1.0437	3.4935	1.0493	3.3011	1.0553	3.1298	1.0617	2.0762	22
39	1.0438	3.4901	1.0494	3.2981	1.0554	3.1271	1.0618	2.9738	21
40	1.0438	3.4867	1.0495	3.2951	1.0555	3.1244	1.0619	2.9713	20
41	1.0439	3.4833	1.0496	3.2021	1.0556	3.1217	1.0620	2.9689	10
42	1.0440	3.4799	1.0497	3.2891	1.0557	3.1190	1.0622	2.9665	18
43	1.0441	3.4766	1.0498	3.2861	1.0558	3.1163	1.0623	2.9641	17
44	1.0442	3.4732 3.4698	1.0499	3.2831	1.0559	3.1137	1.0624	2.9617	16
45 46	I.0443 I.0444	3.4665	1.0500	3.2772	1.0560	3.1110	1.0625	2.0503	15
47	1.0445	3.4632	1.0502	3.2742	1.0562	3.1057	1.0627	2.9545	13
48	1.0446	3.4598	1.0503	3.2712	1.0563	3.1030	1.0628	2.9521	12
49	1.0447	3.4565	1.0504	3.2683	1.0565	3.1004	1.0620	2.9497	II
50	1.0448	3.4532	1.0505	3.2053	1.0566	3.0977	1.0630	2.9474	10
51	1.0448	3.4408	1.0506	3.2624	1.0567	3.0951	1.0632	2.9450	9 8
52	1.0449	3.4465	1.0507	3.2504	1.0568	3.0025	1.0633	2.9426	8
5 3	1.0450	3.4432	1.0508	3.2565	1.0569	3.0898	1.0634	2.9402	7 6
54	1.0451	3.4399 3.4366	1.0509	3.2535	1.0570	3.0872	1.0635	2.9379	
55 56	1.0452	3.4300	1.0510	3.2477	1.0571	3.0820	1.0030	2.9355 2.9332	5 4
	1.0454	3.4301	1.0512	3.2448	1.0573	3.0703	1.0638	2.9332	3
57 58	1.0455	3.4268	1.0513	3.2419	1.0574	3.0767	1.0630	2.0285	2
59	1.0456	3.4236	1.0514	3.2390	1.0575	3.0741	1.0641	2.9261	T
60	1.0457	3.4203	1.0515	3.2361	1.0576	3.0715	1.0642	2 9238	0
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	7 3	30	72	20	71	ا تعد	70)° ŠEC.	

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1 1.0643 2.0215		SEC.	Co-sec.	SEC.	Co-sec.	SEC.	Co-sec.			′
1.0044 2.0101 1.0714 2.7862 1.0788 2.0656 1.0866 2.5586 \$8 3 1.0645 2.0168 1.0715 2.7841 1.0796 2.0618 1.0860 2.5532 56 1.0640 2.0145 1.0717 2.7850 1.0790 2.0618 1.0860 2.5532 56 1.0640 2.0908 1.0710 2.7778 1.0702 2.0500 1.0870 2.5532 56 1.0640 2.0908 1.0710 2.7778 1.0703 2.0580 1.0872 2.5805 6 1.0645 2.0908 1.0710 2.7778 1.0703 2.0580 1.0872 2.5848 34 7 1.0660 2.0952 1.0721 2.7736 1.0704 2.0551 1.0873 2.4871 35 9 1.0653 2.0900 1.0723 2.7694 1.0705 2.0532 1.0874 2.5435 35 9 1.0653 2.0900 1.0723 2.7694 1.0709 2.0532 1.0876 2.5430 51 1.0653 2.0900 1.0723 2.7694 1.0709 2.0535 1.0878 2.5422 40 12.0555 2.8000 1.0726 2.7653 1.0010 2.0406 1.0887 2.5304 41 1.0654 2.8033 1.0726 2.7693 1.0010 2.0406 1.0880 2.5384 48 13 1.0958 2.8037 1.0727 2.7032 1.0802 2.0447 1.0881 2.5307 47 41 1.0658 2.8032 1.0720 2.7501 1.0803 2.0448 1.0882 2.5333 45 15 1.0659 2.8802 1.0732 2.7550 1.0804 2.0410 1.0886 2.5333 45 15 1.0600 2.8806 1.0731 2.7550 1.0804 2.0410 1.0886 2.5333 45 15 1.0600 2.8856 1.0731 2.7590 1.0804 2.0410 1.0886 2.5303 47 41 1.0661 2.8846 1.0733 2.7550 1.0805 2.0335 1.0880 2.5204 41 1.0666 2.8858 1.0732 2.7488 1.0811 2.0316 1.0880 2.5204 41 1.0666 2.8858 1.0732 2.7488 1.0811 2.0316 1.0801 2.5335 43 1.0666 2.8756 1.0731 2.7427 1.0815 2.0200 1.0803 2.5203 33 1.0666 2.8756 1.0737 2.7488 1.0815 2.0200 1.0803 2.5203 33 1.0666 2.8756 1.0737 2.7488 1.0815 2.0200 1.0800 2.5203 33 1.0666 2.8756 1.0737 2.7495 1.0815 2.0200 1.0803 2.5217 30 1.0663 2.8777 1.0747 2.7325 1.0850 2.0508 1.0000 2.5203 3.0904 2.5203 3.0906 2.5203 3.0906 2.5203 3.0906 2.5203 3.0906										
3 1.0645 2.9168					2.7383				2.5575	59
4 1.0646 2.9145							2.0050			
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6 1.6048 2.9008 1.0710 2.7778 1.0793 2.6580 1.0872 2.5481 53 8 1.6051 2.9052 1.0720 2.7757 1.0790 2.05561 1.0873 2.5471 53 8 1.6051 2.9052 1.0721 2.7735 1.0795 2.0542 1.0874 2.5453 52 91.0652 2.0920 1.0722 2.7715 1.0795 2.05542 1.0874 2.5453 52 91.0652 2.0920 1.0722 2.7715 1.0795 2.05542 1.0874 2.5453 52 91.0726 2.7715 1.0795 2.05542 1.0877 2.5419 50 11 1.0653 2.9000 1.0723 2.7694 1.0790 2.0465 1.0877 2.5419 50 11 1.0655 2.8060 1.0726 2.0553 1.0801 2.0460 1.0885 2.5584 41 1.0725 2.0553 1.0801 2.0460 1.0885 2.5584 41 1.0725 2.0553 1.0801 2.0460 1.0885 2.5584 41 1.0727 2.7032 1.0802 2.0447 1.0881 2.5307 47 14 1.0656 2.8093 1.0727 2.7032 1.0803 2.0447 1.0881 2.5307 47 14 1.0656 2.8806 1.0731 2.7570 1.0803 2.0428 1.0852 2.5333 45 15 1.00502 2.8802 1.0728 2.7550 1.0807 2.0372 1.0886 2.5534 41 1.0730 2.7550 1.0806 2.0591 1.0855 2.5333 45 15 1.0660 2.8806 1.0731 2.7570 1.0806 2.0531 1.0885 2.5251 4.0730 2.7550 1.0807 2.0372 1.0886 2.5204 41 1.0664 2.8788 1.0733 2.7529 1.0808 2.0333 1.0880 2.5204 41 1.0664 2.8788 1.0733 2.7529 1.0808 2.0333 1.0880 2.5204 41 1.0664 2.8778 1.0730 2.7488 1.0812 2.0207 1.0891 2.5247 40 1.0666 2.8856 1.0731 2.7427 1.0851 2.0310 1.0891 2.5247 40 1.0666 2.8856 1.0731 2.7427 1.0811 2.0310 1.0891 2.5247 40 1.0660 2.8688 1.0731 2.7427 1.0815 2.0207 1.0802 2.5233 38 1.0730 2.7447 1.0815 2.0207 1.0802 2.5233 38 1.0730 2.7447 1.0815 2.0207 1.0802 2.5233 38 1.0730 2.7447 1.0815 2.0207 1.0802 2.5233 38 1.0730 2.7447 1.0815 2.0207 1.0802 2.5210 3.0800 2.5110 3.080 2.5210 3.0800 2.5110 3.0801 2.0808 2.0807 2.5103 38 1.0070 2.8688 1.0744 2.7346 1.0817 2.0223 1.0800 2.5110 3.080 2.5210 3.0800 2.5110 3.0801 2.0808 2.0808 1.0071 2.8644 1.0743 2.7366 1.0817 2.0223 1.0800 2.5110 3.0801 2.0808 1.0071 2.8644 1.0748 2.7386 1.0822 2.0056 1.0900 2.5123 38 1.0070 2.8638 1.0749 2.7385 1.0826 2.0055 1.0900 2.5123 38 1.0070 2.8488 1.0749 2.7385 1.0826 2.0056 1.0900 2.5123 38 1.0070 2.8488 1.0074 2.8581 1.0748 2.7385 1.0826 2.0000 1.0000 2.5123 38 1.0070 2.8488 1.0074 2.7385 1.0828 2.0000 1.0000 2.5123 38 1.0000 2.848										
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13						1.0799			2.5402	49
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2	1.0949	2.4554	1.1037	2.3632	1.1129	2.2784	1.1226	2.2002 2.198 9	58
3 4	1.0951	2.4530	1.1030	2.3603	1.1131	2.2757	1.1220	2.1959	57 56
	1.0953	2.4506	1.1041	2.3588	1.1134	2.2744	1.1231	2.1964	55
5 6	1.0955	2.4490	1.1043	2.3574	1.1135	2.2730	1.1233	2.1952	54
7 8	1.0956	2.4474	1.1044	2.3559	1.1137	2.2717	1.1235	2.1939	53
	1.0058	2.4458	1.1046	2.3544	1.1139	2.2703	1.1237	2.1927	52
9	1.0959	2.4442	1.1047	2.3530	1.1140	2.2690	1.1238	2.1914	51
10	1.0001	2.4426	1.1049	2.3515	1	1		2.1902	50
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13	1.0003	2.4379	1.1052	2.3472	1.1147	2.2636	1.1245	2.1865	47
14	1.0966	2.4363	1.1055	2.3457	1.1148	2 2623	1.1247	2.1852	46
15	1.0968	2.4347	1.1056	2.3443	1.1150	2.2610	1.1248	2.1840	45
16	1.0069	2.4332	1.1058	2.3428	1.1151	2.2596	1.1250	2.1828	44
17	1.0971	2.4316	1.1059	2.3414	1.1153	2.2583	1.1252	2.1815	43
18	1.0972	2.4300	1.1061	2.3399	1.1155	2.2570	1.1253	2.1803	42
19	1.0973	2.4285	1.1062	2.3385	1.1156	2.2556	1.1255	2.1791	41
20	1.0975	2.4200	1.1064	2.3371	1.1158	2.2543	1.1257	2.1778	40
21	1.0976	2.4254	1.1065	2.3356	1.1159	2.2530	1.1258	2.1766	39
22	1.0078	2.4238	1.1067	2.3342	1.1161	2.2517	1.1260	2.1754	38
23 24	1.0979	2.4222	1.1068	2.3328	1.1163	2.2503	1.1202	2.1742	37 36
25	1.0082	2.4101	1.1073	2.3299	1.1166	2.2477	1.1265	2.1717	35
26	1.0984	2.4176	1.1073	2.3285	1.1167	2.2464	1.1267	2.1705	34
27	1.0985	2.4160	1.1075	2.3271	1.1169	2.2451	1.1269	2.1693	33
28	1.0986	2.4145	1.1076	2.3256	1.1171	2.2438	1.1270	2.1681	32
20	1.0988	2.4130	1.1078	2.3242	1.1172	2.2425	1.1272	2.1669	31
30	1.0989	2.4114	1.1079	2.3228	1.1174	2.2411	1.1274	2.1657	30
31	1.0991	2.4099	1.1081	2.3214	1.1176	2.2398	1.1275	2.1645	20
32	1.0002	2.4083	1.1082	2.3200	1.1177	2.2385	1.1277	2.1633	28
33	1.0004	2.4068	1.1084	2.3186	1.1179	2.2372	1.1279	2.1620	27
34	1.0005	2.4053	1.1085	2.3172	1.1180	2.2359	1.1281	2.1608	26
35 36	1.0008	2.4037	1.1088	2.3158	1.1184	2.2333	1.1284	2.1584	24
37	1.1000	2.4007	1.1000	2.3129	1.1185	2.2320	1.1286	2.1572	23
38	1.1001	2.3002	1.1002	2.3115	1.1187	2.2307	1.1287	2.1560	22
39	1.1003	2.3976	1.1003	2.3101	1.1189	2.2294	1.1289	2.1548	21
40	1.1004	2.3961	1.1095	2.3087	1.1190	2.2282	1.1291	2.1536	20
41	1.1005	2.3946	1.1096	2.3073	1.1192	2.2269	1.1293	2.1525	19
42	1.1007	2.3931	1.1098	2.3059	1.1193	2.2256	1.1204	2.1513	18
43	1.1008	2.3916	1.1099	2.3046	1.1195	2.2243	1.1206	2.1501	17
44	1.1010	2.3001	1.1101	2.3032	1.1197	2.2230	1.1298	2.1489	16
45 46	1.1011	2.3886	1.1102	2.3018	1.1198	2.2217	1.1200	2.1477	15
47	1.1013	2.3856	1.1104	2.2000	1.1202	2.2102	1.1303	2.1453	13
48	1.1016	2.3841	1.1107	2.2076	1.1203	2.2170	1.1305	2.1441	12
49	1.1017	2.3826	1.1109	2.2962	1.1205	2.2166	1.1306	2.1430	11
50	1.1010	2.3811	1.1110	2.2949	1.1207	2.2153	1.1308	2.1418	10
51	1.1026	2.3796	1.1112	2.2935	1.1208	2.2141	1.1310	2.1406	9 8
52	I 1022	2.3781	1.1113	2.2921	I 1210	2.2128	1.1312	2.1394	8
53	1.1023	2.3766	1.1115	2.2907	1.1212	2.2115	1.1313	2.1382	7 6
54	1.1025	2.3751	1.1116	2.2894	1.1213	2.2103	1.1315	2.1371	
55	1.1026	2.3736	1.1118	2.2880	1.1215	2.2000	1.1317	2.1359 2.1347	5 4
56 57	1.1020	2.3721	1.1120	2.2853	1.1217	2.2005	1.1320	2.1347	3
58	1.1029	2.3691	1.1123	2.2830	1.1220	2.2052	1.1322	2.1324	2
59	1.1032	2.3677	1.1124	2.2825	1.1222	2.2030	1.1324	2.1312	1
60	1.1034	2.3662	1.1126	2.2812	1.1223	2.2027	1.1326	2.1300	0
	Coara	Spa	Coare	Sec	Co-crec	SEC.	Co-sec.	SEC.	-
	Co-sec.	SEC.	Co-sec.	SEC.	Co-sec	3° SEC.	62	20 25.	

	2	8°	1 29	90	1 30)°	31°		
,	SEC.	Co-sec.	SEC.	Co-sec.	SEC.	Co-sec.	SEC.	Co-sec.	,
0	1.1326	2.1300	1.1433	2.0627	1.1547	2.0000 I.0000	1.1666	1.9416	60
2	1.1327	2.1277	1.1437	2.0605	1.1549	1.0080	1.1670	1.9397	5 9 58
3	1.1331	2.1200	1.1439	2.0594	1.1553	1.9970	1.1672	1.9388	57
4	1.1333 1.1334	2.1254	1.1441	2.0583	1.1555	1.9960	1.1674	1.9378	56
5 6	1.1336	2.1231	1.1445	2.0562	1.1559	1.9940	1.1678	1.0300	55 54
7 8	1.1338	2.1210	1.1446	2.0551	1.1561	1.9930	1.1681	1.9350	53
9	1.1340	2.1208 2.1196	1.1448	2.0540	1.1562	1.0020	1.1683	1.9341	52 51
10	1.1343	2.1185	1.1452	2.0519	1.1566	1.9900	1.1687	1.9332	50
11	1.1345	2.1173	1.1454	2.0508	1.1568	1.9890	1.1689	1.9313	49
12	1.1347	2.1162	1.1456	2.0498	1.1570	1.1570 1.9880		1.0304	48
13	1.1349	2.1150	1.1450	2.0407	1.1572 1.9870 1.1574 1.9860		1.1693	1.0205	47 46
15	1.1352	2.1127	1.1461	2.0466	1.1576	1.9850	1.1697	1.9276	45
16	1.1354	2.1116	1.1463	2.0455	1.1578	1.9840	1.1699	1.9267	44
17 18	1.1356	2.1104	1.1465	2.0444	1.1580	1.9830	1.1701	1.9258	43 42
19	1.1359	2.1082	1.1469	2.0423	1.1584	1.9811	1.1705	1.9239	41
20	1.1361	2.1070	1.1471	2.0413	1.1586	1.9801	1.1707	1.9230	40
2 I 2 2	1.1363	2.1059	1.1473	2.0402	1.1588	1.9791	1.1709	1.9221 1.9212	39 38
23	1.1366	2.1036	1.1476	2.0381	1.1592	1.9771	1.1714	1.9203	37
24	1.1368	2.1025	1.1478	2.0370	1.1594	1.9761	1.1716	1.9193	36
25 26	1.1370 1.1372	2.1014 1.148c 2.1002 1.1482		2.0360	1.1596	1.9752 1.9742	1.1718	1.9184	35 34
27	1.1373	2.0991	1.1484	2.0339	1.1600	1.9732	1.1722	1.9166	33
28	1.1375	2.0980	1.1486	2.0329	1.1602	1.9722	1.1724	1.9157	32
29 30	1.1377	2.0969	1.1488	2.0318	1.1604	1.9713	1.1726	1.9148 1.9139	3I 30
31	1.1381	2.0946	1.1491	2.0297	1.1608	1.9693	1.1730	1.0130	29
32	1.1382	2.0935	1.1493	2.0287	1.1610	1.9683	1.1732	1.0121	28
33 34	1.1386	2.0012			1.1612	1.9674 1.9664	1.1734	1.9112	27 26
35	1.1338	2.0901	1.1499 2.0256		1.1616	1.9654	1.1739	1.0003	25
36 37	1.1390	2.0890			1.1618	1.9645 1.9635	1.1741	1.9084	24
38	1.1393	2.0868	1.1505	2.0235	1.1622	1.9625	1.1743	1.90/5	23
39	1.1395	2.0857	1.1507	2.0214	1.1624	1.9616	1.1747	1.9057	2 I
40	1.1397	2.0846	1.1508	2.0204	1.1626	1.9606	1.1749	1.9048	20
41 42	1.1399	2.0835	1.1510	2.0194	1.1628	1.9596	1.1751	1.9039	18
43	1.1402	2.0812	1.1514	2.0173	1.1632	1.9577	1.1756	1.9021	17
44	1.1404	2.0801	1.1516	2.0163	1.1634	1.9568	1.1758	1.9013	16
45 46	1.1406	2.0790	1.1518	2.0152	1.1636	1.9558 1.9549	1.1760	1.0004	15
47	1.1410	2.0768	1.1522	2.0132	1.1640	1.9539	1.1764	1.8086	13
48 49	1.1411	2.0757	1.1524	2.0122	1.1642	1.9530 1.9520	1.1766	1.8977 1.8968	I2
50	1.1415	2.0735	1.1528	2.0101	1.1646	1.9510	1.1770	1.8959	10
51	1.1417	2.0725	1.1530	2.0091	1.1648	1.9501	1.1772	1.8950	9 8
52	1.1419	2.0714	1.1531	2.0081	1.1650	1.9491	1.1775	1.8941	8
53 54	1.1421	2.0703	1.1533	2.0071	1.1652	1.9482	1.1777	1.8032	7 6
55	1.1424	2.0681	1.1537	2.0050	1.1656	1.9463	1.1781	1.8915	5
56	1.1426	2.0670	1.1539	2.0040	1.1658	I.9454 I.9444	1.1783	1.8906	4 2
57 58	1.1430	2.0648	1.1543	2.0030	1.1662	1.9444	1.1787	1.8888	3 2
59 60	1.1432	2.0637	1.1545	2.0010	1.1664	1.9425	1.1790	1.8879	I
	1.1433	2.0627	1.1547	2.0000	1.1666	1.9416	1.1792	1.8871	0
,	Co-sec.	SEC.	Co-sec.	SEC.	Co-sec.	SEC.	Co-sec.	SEC.	1
	61	ا ۱	60)	II 59	9°	58°		

	32) e c	33° II		1 34	10 !	35° I		
,	SEC.	Co-sec.	SEC.	Co-sec.				Co-sec.	,
0	1.1792	1.8871	1.1924	1.8361	1.2062	1.7883	1.2208	1.7434	 6a
I	1.1794	1.8862	1.1926	1.8352		1.2064 1.7875		1.7427	59
2	1.1796	1.8853	1.1028	1.8344	1.2067	1.7867	1.2210	1.7420	58
3	1.1798	1.8844	1.1930	1.8336	1.2060	1.7860	1.2215	1.7413	57
4	1.1800	1.8836	1.1933	1.8328	1.2072 1.7852		1.2218	1.7405	56
	1.1802	1.8827	1.1935	1.8320	1.2074			1.7398	55
5 6	1.1805	1.8818	1.1937	1.8311	1.2076	1.7837	1.2220	1.7391	54
	1.1807	1.8800	1.1939	1.8303	1.2079	1.7829	1.2225	1.7384	53
7 8	1.1800	1.8801	1.1042	1.8295	1.2081	1.7821	1.2228	1.7377	52
9	1.1811	1.8702	1.1044	1.8287	1.2083	1.7814	1.2230	1.7360	51
10	1.1813	1.8783	1.1946	1.8279	1.2086	1.7806	1.2233	1.7362	50
11	1.1815	1.8785	1.1948	1.8271	1.2088	1.7798	1.2235	1.7355	
	1.1818	1.8766	1.1940	1.8263	1.2001	1.7791	1.2238	1.7348	49 48
12	1.1820		1.1953	1.8255			1.2240		
13	1.1822	1.8757	1.1953	1.8246	1.2093 1.7783		1.2243	1.7341	47
14	1.1824	1.8749	1.1958	1.8238	1.2008	1.7768	1.2243	1.7334	46
15	1.1824	1.8731	1.1060	1.8230	1.2093	1.7760	1.2248	1.7327	45
16	1.1828	1.8723	1.1962	1.8230	1.2103	1.7753	1.2240	1.7319	44
17	1.1831	1.8714	1.1964	1.8214	1.2103	1.7745	1.2253	1.7312	43 42
			1.1904	1.8206	1.2105		1.2255	1.7305	42 41
19	1.1833	1.8706	1.1969	1.8198	1.2110	1.7738	1.2258	1.7298	40
20		1	1 ' '		1!		-		
21	1.1837	1.8688	1.1971	1.8190	1.2112	1.7723	1.2260	1.7284	39
22	1.1839	1.8680	1.1974	1.8182	1.2115	1.7715	1.2263	1.7277	38
23	1.1841	1.8671	1.1976	1.8174	1.2117	1.7708	1.2265	1.7270	37
24	1.1844	1.8663	1.1978	1.8166	1.2119	1.7700	1.2268	1.7263	36
25	1.1846	1.8654	1.1980	1.8158	1.2122	1.7093	1.2270	1.7256	35
26	1.1848	1.8646	1.1983	1.8150	1.2124	1.7685	1.2273	1.7249	34
27.	1.1850	1.8637	1.1985	1.8142	1.2127	1.7678	1.2276	1.7242	33
28	1.1852	1.8629	1.1987	1.8134	1.2129	1.7670	1.2278	1.7234	32
29	1.1855	1.8620	1.1990	1.8126	1.2132	1.7663	1.2281	1.7227	31
30	1.1857	1.8611	1.1992	1.8118	1.2134	1.7655	1.2283	1.7220	30
31	1.1850	1.8603	1.1994	1.8110	1.2136	1.7648	1.2286	1.7213	20
32	1.1861	1.8505	1.1997	1.8102	1.2130	1.7640	1.2288	1.7206	28
33	1.1863	1.8586	1.1999	1.8094	1.2141	1.7633	1.2291	1.7199	27
34	1.1866	1.8578	1.2001	1.8086	1.2144	1.7625	1.2293	1.7192	26
35	1.1868	1.8569	1.2004	1.8078	1.2146	1.7618	1.2296	1.7185	25
36	1.1870	1.8561	1.2006	1.8070	1.2149	1.7610	1.2298	1.7178	24
37	1.1872	1.8552	1.2008	1.8062	1.2151	1.7603	1.2301	1.7171	23
3 8	1.1874	1.8544	1.2010	1.8054	1.2153	1.7596	1.2304	1.7164	22
39	1.1877	1.8535	1.2013	1.8047	1.2156	1.7588	1.2306	1.7157	21
40	1.1879	1.8527	1.2015	1.8039	1.2158	1.7581	1.2309	1.7151	20
41	1.1881	1.8510	1.2017	1.8031	1.2161	1.7573	1.2311	1.7144	10
42	1.1883	1.8510	1.2020	1.8023	1.2163	1.7566	1.2314	1.7137	18
43	1.1886	1.8502	1.2022	1.8015	1.2166	1.7559	1.2316	1.7130	17
44	1.1888	1.8493	1.2024	1.8007	1.2168	1.7551	1.2319	1.7123	16
45	1.1800	1.8485	1.2027	1.7999	1.2171	1.7544	1.2322	1.7116	15
46	1.1802	1.8477	1.2020	1.7002	1.2173	1.7537	1.2324	1.7109	14
47	1.1894	1.8468	1.2031	1.7984	1.2175	1.7529	1.2327	1.7102	13.
48	1.1897	1.8400	1.2034	1.7976	1.2178	1.7522	1.2320	1.7005	12
49	1.1800	1.8452	1.2036	1.7068	1.2180	1.7514	1.2332	1.7088	11
50	1.1001	1.8443	1.2039	1.7960	1.2183	1.7507	1.2335	1.7081	10
51	1.1003	1.8435	1.2041	1.7953	1.2185	1.7500	1.2337	1.7075	9
52	1.1006	1.8427	1.2043	1.7945	1.2188	1.7403	1.2340	1.7068	- 8
53	1.1008	1.8418	1.2046	1.7937	1.2100	1.7485	1.2342	1.7061	7
54	1.1910	1.8410	1.2048	1.7929	1.2193	1.7478	1.2345	1.7054	6
55	1.1012	1.8402	1.2050	1.7921	1.2195	1.7471	1.2348	1.7047	5
56	1.1015	1.8394	1.2053	1.7914	1.2108	1.7463	1.2350	1.7040	4
57	1.1017	1.8385	1.2055	1.7006	1.2200	1.7456	1.2353	1.7033	3
58	1.1010	1.8377	1.2057	1.7808	1.2203	1.7449	1.2355	1.7027	2
59	1.1921	1.8369	1.2060	1.7891	1.2205	1.7442	1.2358	1.7020	1
60	1.1922	1.8361	1.2062	1.7883	1.2208	1.7434	1.2361	1.7013	0
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•	Co-sec.	SEC.	Co-sec.	SEC.	Co-sec.	SEC.	Co-sec.	SEC.	
	1 5	7°	11 5	6°	11 5	5°	u 5	40	1

	36		37	(0 (38	30 ii	39		
'	SEC.	Co-sec.	SEC.	Co-sec.	SEC.			Co-sec.	,
0	1.2361	1.7013	1.2521	1.6616	1.2690 1.2693	1.6243	1.2867	1.58go 1.5884	60 59.
3	1.2366	1.6999 1.6993	1.2527	1.6603	1.2696 1.2699	1.6231	1.2874	1.5879	58 57
4 5 6	1.2371	1.6986	1.2532	1.6591	1.2702	1.6218	1.2880	1.5867	56 55
6 7 8	1.2376	1.6972	1.2538	1.6578	1.2707 1.2710	1.6206	1.2886	1.5856 1.5850	54 53
8	1.2382	1.6959	1.2543 1.2546	1.6565	1.2713	1.6194	1.2892	1.5845	52 51
10	1.2387	1.6945	1.2549	1.6552	1.2719	1.6182	1.2898	1.5833	50 49
12	1.2392	1.6932	1.2554	1.6540	1.2725	1.6170	1.2904	1.5822	48
14	1.2397	1.6918	1.2560	1.6527	1.2731	1.6159	1.2910	1.5811	46
16 17	1.2403	1.6905	1.2565	1.6514	1.2737	1.6147	1.2016	1.5799	44 43
18 10	1.2408	1.6891	1.2571	1.6502	1.2742	1.6135	1.2022	1.5788	43 42 41
20	1.2413	1.6878	1.2577	1.6489	1.2748	1.6123	1.2929	1.5777	40
2 I 2 2	1.2415	1.6865	1.2579	1.6483 1.6477	1.2751	1.6117	1.2932	1.5771 1.5766	3 9 38
· 23	1.2421	1.6858	1.2585	1.6470	1.2757	1.6105	1.2938	1.5750	37 36
25 26	1.2427	1.6845	1.2591	1.6458	1.2763 1.2766	1.6087	1.2944	1.5749 1.5743	3 5 34
27 28	1.2432	1.6831	1.2596	1.6445	1.2709	1.6081	1.2953	1.5738	33 32
29 30	1.2437	1.6818	1.2602	1.6433 1.6427	1.2775	1.6070	1.2956	1.5727 1.5721	31 30
31 32	1.2443	1.6805	1.2607	1.6420	1.2781	1.6058	1.2963	1.5716	29 28
33 34	1.2448	1.6792	1.2613	1.6408	1.2787	1.6046	1.2969	1.5705	27 26
35 36	1.2453	1.6779	1.2619	1.6396	1.2793	1.6034	1.2975	1.5694	25 24
37 38	1.2459	1.6766	1.2624	1.6383	1.2798	1.6023	1.2981	1.5683	23
39 40	1.2464	1.6752	1.2630	1.6371	1.2804	1.6011	1.2988	1.5672	2 I 20
41 42	1.2470	1.6739 1.6733	1.2636 1.2639	1.6359	1.2810	1.6000	1.2994	1.5661	19
43 44	1.2475	1.6726	1.2641	1.6346	1.2816	1.5988	1.3000	1.5650	17
45 46	1.2480	1.6713	1.2647	1.6334	1.2822	1.5976	1.3006	1.5639	15
47 48	1.2486	1.6700	1.2653	1.6322	1.2828	1.5965	1.3013	1.5628	13
49 ,0	1.2490	1.6687	1.2659	1.6309	1.2834	1.5953	1.3010	1.5617	11
51	1.2497	1.6674	1.2664	1.6297	1.2840	1.5942	1.3025	1.5606	9 8
52 53	1.2499	1.6668	1.2667	1.6291	1.2843	1.5936 1.5930	1.3029	1.5500	7 6
54 55	1.2505	1.6655	1.2673	1.6279	1.2849	1.5924	1.3035	1.5590	5
56 57	1.2513	1.6642	1.2679	1.6267	1.2855	1.5913	1.3041	1.5579	3
58 59	1.2516	1.6623	1.2684	1.6255	1.2861	1.5901	1.3048	1.5568	1 1
60	1.2521	1.6616	1.2690	1.6243 SEC.	1.2867	1.5890	1.3054	1.5557 SEC.	-
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2	1.3060	1.5546	1.3257	1.5232	1.3463	1.4935	1.3681	1.4654	58
3	1.3064	1.5541	1.3260	1.5227	1.3467	1.4930	1.3684	1.4649	57
4	1.3067	1.5536			1.3470	1.4925 1.4921	1.3688	1.4644	56 55
5	1.3073	1.5525			1.3477	1.4916	1.3605	1.4635	54
7 8	1.3076	1.5520	1.3274	1.5207	1.3481	1.4011	1.3699	1.4631	53
	1.3080	1.5514	1.3277	1.5202	1.3485	1.4906	1.3703	1.4626	52
9	1.3083	1.5509 1.5503	1.3280	1.5197	1.3400	1.3488 1.4901 1.3492 1.4897		1.4622 1.4617	51 50
11	1.3080	1.5498	1.3287	1.5187	1.3495	1.4802	1.3710	1.4613	49
12	1.3002	I 5493	1.3290	1.5182	1.3499	1.4887	1.3718	1.4608	48
13	1.3000	1.5487	1.3294	1.5177	1.3502	1.4882	1.3722	1.4604	47
14	1.3099	1.5482	1.3207	1.5171	1.3506	1.4877	1.3725	1.4599	46
16	1.3102	I.5477 I.547I	1.3301	1.5166	1.3509	1.4873 1.4868	1.3729	1.4595	45 44
17	1.3100	1.5466	1.3307	1.5156	1.3517	1.4863	1.3737	1.4586	43
18	1.3112	1.5461	1.3311	1.5151	1.3520	1.4858	1.3740	1.4581	42
19	1.3115	1.5456	1.3314	1.5146	1.3524	1.4854	1.3744	1.4577	41
20	1.3118	1.5450	1.3318	1.5141	1.3527	1.4849	1.3748	1.4572	40
2I 22	1.3121	1.5445	1.3321	1.5136	1.3531	1.4844	1.3752 1.3756	1.4568 1.4563	39 38
23	1.3128	1.5434	1.3324	1.5126	1.3534	1.4835	1.3759	1.4559	37
2.1	1.3131	1.5429	1.3331	1.5121	1.3542	1.4830	1.3763	1.4554	36
25	1.3134	1.5424	1.3335	1.5116	1.3545	1.4825	1.3767	1.4550	3 5
20	1. 138	1.5419	1.3338	1.5111	1.3549	1.4821	1.3771	1.4545	34
27 28	1.3141	1.5413	I.3342 I.3345	1.5101	1.3552	1.4811	1.3774	1.4541	33 32
29	1.3148	1.5403	1.3348	1.5096	1.3560	1.4806	1.3782	1.4532	31
30	1.3151	1.5398	1.3352	1.5092	1.3563	1.4802	1.3786	1.4527	30
31	1.3154	1.5392	1.3355	1.5087	1.3567	1.4707	1.3790	1.4523	20
32	1.3157	1.5387	1.3359	1.5082	1.3571	1.4702	1.3794	1.4518	28
33 34	1.3161	1.5382	1.3362	1.5077	1.3574	1.4788	1.3797	1.4514	27 26
35	1.3167	1.5371	1.3369	1.5067	1.3581	1.4778	1.3805	1.4505	25
36	1.3170.	1.5366	1.3372	1.5062	1.3585	1.4774	1.3809	1.4501	24
37	1.3174	1.5361	1.3376	1.5057	1.3589	1.4769	1.3813 1.3816	1.4496	23
38 39	1.3177	1.5356	1.3379 1.5053 1.3383 1.504		1.3592	1.4760	1.3820	1.4487	21
40	1.3184	1.5345	1.3386	1.5042	1.3600	1.4755	1.3824	1.4483	20
41	1.3187	1.5340	1.3390	1.5037	1.3603	1.4750	1.3828	1.4479	19
42	1.3190	1.5335	1.3393	1.5032	1.3607	1.4746	1.3832	1.4474	18
43	1.3193	1.5330	1.3397	1.5027	1.3611	1.4741	1.3836	1.4470	17 16
44 45	1.3197	1.5325	1.3400	1.5022	1.3614	I.4736 I.4732	1.3839 1.3843	1.4465	15
46	1.3203	1.5314	1.3407	1.5013	1.3622	1.4727	1.3847	1.4457	14
47	1.3207	1.5309	1.3411	1.5008	1.3625	1.4723	1.3851	1.4452	13
48	1.3210	1.5304	1.3414	1.5003	1.3629	1.4718	1.3855	1.4448	I 2 I I
49 50	1.3213	1.5299	1.3418	1.4998	1.3633	1.4713	1.3863	I.4443 I.4439	10
51	1.3220	1.5289	1.3425	1.4988	1.3640	1.4704	1.3867	1.4435	
52	1.3223	1.5283	1.3428	1.4983	1.3644	1.4699	1.3870	1.4430	9 8
. 53	1.3227	1.5278	1.3432	1.4979	1.3647	1.4695	1.3874	1.4426	7 6
54	1.3230	1.5273	1.3435	1.4974	1.3651	1.4690	1.3878	1.4422	٥
55 56	1.3233	1.5268	1.3439	1.4969	1.3655	1.4681	1.3886	1.4417	5 4
57	1.3240	1.5258	1.3446	1.4959	1.3662	1.4676	1.3890	1.4408	3
58	1.3243	1.5253	1.3449	1.4954	1.3666	1.4672	1.3894	1.4404	2
50 60	1.3247	1.5248	1.3453	1.4949	1.3669	1.4663	1.3898	1.4400	0
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ī	1.3005	1.4391	59	22	1.3988	1.4301	38	42	1.4069	1.4217	18
2	1.3909	1.4387	58	23	1.3992	1.4297	37	43	1.4073	1.4212	17
3	1.3913	1.4382	57	2.4	1.3996	1.4292	36	44	1 4077	1.4208	16
4	1.3917	1.4378	56	25	1.4000	1.4288	35	45	1.4081	1.4204	15
5 6	1.3921	1.4374	55	26	1.4004	1.4284	34	46	1.4085	1.4200	14
	1.3925	1.4370	54	27	1.4008	1.4280	33	47	1.4089	1.4196	13
7 8	1.3929	1.4365	53	28	1.4012	1.4276	32	48	1.4093	1.4192	12
8	1.3933	1.4361	52	29	1.4016	1.4271	31	49	1.4097	1.4188	11
9	1.3937	1.4357	51	30	1.4020	1.4267	30	50	1.4101	1.4183	10
10	1.3941	1.4352	50	31	1.4024	1.4263	29	51	1.4105	1.4179	9 8
11	1.3945	1.4348	49	32	1.4028	1.4259	28	52	1.4109	1.4175	
12	1.3949	1.4344	48	33	1.4032	1.4254	27	53	1.4113	1.4171	7
13	1.3953	1.4339	47	34	1.4036	1.4250	26	54	1.4117	1.4167	
14	1.3957	1.4335	46	35	1.4040	1.4246	25	55	1.4122	1.4163	5
15	1.3960	1.4331	45	36	1.4044	1.4242	24	56	1.4126	1.4159	4.
16	1.3964	1.4327	44	37	1.4048	1.4238	23	57	1.4130	1.4154	3
17	1.3968	1.4322	43	38	1.4052	1.4233	22	58	1.4134	1.4150	2
18	1.3972	1.4318	42	39	1.4056	1.4229	21	59	1.4138	1.4146	1
19	1.3976	1.4314	41	40	1.4060	1.4225	20	60	1.4142	1.4142	c
20	1.3980	1.4310	40								
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DICTIONARY OF MACHINE SHOP TERMS

This has been compiled to assist in a definite understanding of the names of tools, appliances and shop terms which are used in various parts of the country, and will, we trust, prove of value in this way. Cross references have been used in many cases, and we believe that no trouble will be experienced in finding the definition desired even where it may not be under the letter expected. Cutters of all kinds are under "cutters," twist drills under "drills," and by bearing this in mind no delay will be experienced. Practical suggestions as to additions to this section will be appreciated.

DICTIONARY OF SHOP TERMS

Ampere - The unit of electric current. The amount of currers which one volt can force through a resistance of one ohm.

Ampere Hour. — One ampere flowing for one hour.

Ampere Turns. - Used in magnet work to represent the number of

turns times the number of amperes.

Angle Irons - Pieces, usually castings, for holding work at an angle with the face-plate of a lathe, the platen of a planer or other similar work. Usually at right angles but can be anything desired.

Angle Plate - A cast-iron plate with two surfaces at right angles to each other; one side is bolted to a machine table, the other carries

the work.

Annealing - Softening steel, rolled brass or copper by heating to a low heat and allowing to cool gradually.

Annealing Boxes — Boxes, usually of cast iron, in which steel is packed with lime or sand to retard the cooling as much as possible. Anode — The positive terminal of any source of electricity as a battery, or where the current goes into a plating bath.

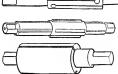
Anvils — Blocks of iron or steel on which metals are hammered or forged. Usually have a steel face. A square hole is usually provided for holding hardies, fuller blocks, etc.

Apron — A protecting or covering piece which encloses or covers any

mechanism, as the apron of a lathe.

Arbor - Shaft or bar to hold work while it is being turned or otherwise worked on. Usually made with a slight taper (about .010 inch per foot) to drive into work and hold by friction. Also

applied to shaft for holding circular saws, milling cutters, etc. Often called mandrel.



Arbor, Expansion — Arbor which can be varied in diameter to hold different sized work. These vary greatly in design, as shown. The first and last are spring sleeves of different types, the second has blades which can be adjusted to size.

Arc — The passage of current across the space between two separated points.

Armature. — Usually the revolving part of a dynamo or motor or the movable part of any magnetic device.

Babbitt Metal - A good mixture for bearings where the load is not too heavy. Consists of varying proportions of tin, antimony, and copper, and sometimes lead. Tin is the base.

Back-lash - Usually applied to lost motion in gears, sometimes to

screw in a nut.

Backing-off - Removing metal behind the cutting edge to relieve friction in taps, reamers, drills, etc. Also called "relieving."

Back Rest — A rest attached to the lathe ways for supporting long, slender shafts or other work being turned.

Balance, Running — High-speed pulleys require balancing by running at speed and seeing that they run without tremor or vibration.

This is called running balance.

Balance, Standing - When a pulley has been balanced on the balancing ways it is called a standing balance. See Balance-running.



Balancing Ways - Level strips on which the shaft carrying the pulley or other revolving body is placed. If the pulley is unbalanced the heavy side will roll to bottom.

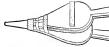
Ball Reamer - See Reamer, Ball.

Bastard - Not regular. The term is usually applied to a file, meaning a cut between the rough and second cut, or to a thread, meaning one that is not of the standard proportions.

Battery. — A combination of chemicals which will give off an electric

Bearings, Ball - Made to reduce friction by interposing balls between the shaft and the bearing. They are made in various ways but all aim to have a rolling instead of a sliding action.

Bearings, Roller - Similar to ball bearings except rollers are used instead of balls. In some cases the rollers are practically hollow round springs from square stock. These are known as flexible roller bearings (Hyatt).



Bellows - Devices of wood and leather for producing a current of air for fanning fires or blowing dust.

Bearing, Base Plate — For supporting pillow blocks or journal boxes. Belt Carriers - Pulleys for supporting a long belt between driving and driven pulleys. May or may not have flanges.

Belt Dressing - Preparation for preserving or cleaning a belt or

making it cling to pulleys.

Belt Fastener - Hooks or other device for joining the ends of belt. Belt Lacing - Methods of fastening ends of belt with a more or less flexible joint by means of leather or wire lacing.

Belt, Muley - A belt running around a corner guided by idler pulleys on a muley shaft.

Belt Polisher or Strap — A belt covered with glue and emery or other abrasive is driven over pulleys and work held against it.

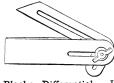
Belt Shifter — Device for shifting belt or belts on countershaft or elsewhere, from loose to tight pulleys and vice-versa. These are made in many varieties. Not used where clutches are employed.

Belt Tightener — Loose pulleys arranged for taking up slack of belts; often called idlers.

Bench — Usual hight is 34 to 35 inches from floor to top of bench, width about 29 inches. Should be 3 inches from wall to allow circulation of air, in order to give sprinklers a chance at a fire underneath.

Bench, Leveling — Bench with a level surface so that work can be laid on it to test. Made of iron.

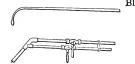
Bending Machine — For bending rods, beams, rails, plate, etc. Run by hydraulic or other power.



Bevel — A tool for measuring or laying off bevels as shown. Also a surface not at right angles to the main surface; may be any angle. When at 45 degrees sometimes called a miter.

Blocks, Differential — Hoisting apparatus consisting of differential gears for lifting heavy loads.

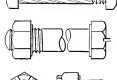
Blocks, Tackle — Sheaves or pulleys mounted in a shell or case, used with hoisting ropes or chains to raise heavy weights.



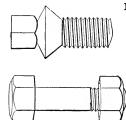
Blow Pipe — A pipe for blowing a jet of air into a flame for heating work locally, such as soldering. The upper picture is a plain one for use with an alcohol lamp, the other has a gas and an air tube. Each is regulated by the small valve so as to make the hottest flame.

BOLTS

Agricultural Bolt. Agricultural bolts, as indicated by the name, are used in farm machines and appliances. The body of the bolt has a series of helical lands and grooves which are formed by a rolling process.



A. L. A. M. Bolt — This bolt is adopted by the Association of Licensed Automobile Manufacturers. It has a slotted head and castellated nut.



Boiler Patch Bolt — A bolt used in fastening patches on boilers. The patch is countersunk for the cone head, and boiler shell tapped for bolt thread. The square head is knocked off after bolt is screwed in place.

Coupling Bolt — Eolts for shaft couplings are finished all over and must be a close fit in the hole reamed in the two flanges of the coupling, so that the sections shall be rigidly secured together.

Expansion Bolt — In attaching parts to brick, stone or concrete walls and floors, expansion bolts are frequently employed. The



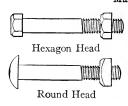
sion bolts are frequently employed. The "Star" bolt in the illustration has an internally threaded, split sleeve which is slipped into a hole made in the wall and then expanded by running in the screw.

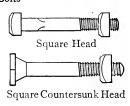
The projections on the surface of the shell, and the fact that the hole receiving it is made larger at the rear, assure the device holding fast when the expander is in place.



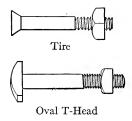
Hanger Bolt — This bolt is used for attaching hangers to woodwork and consists of a lag screw at one end with a machine bolt thread and nut at the other.

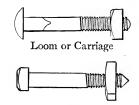
Machine Bolts



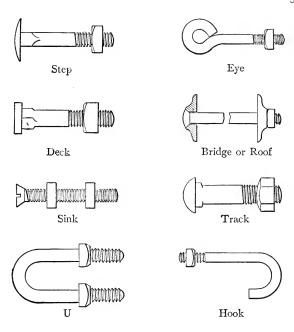


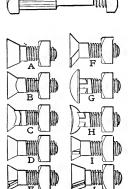
Miscellaneous Bolts





Joint





"North" Bolt — The "North" bolt is used in agricultural machinery and appliances and has a series of longitudinal lands rolled on the body to the same diameter as the bolt.

Plow Bolt — Several types of plow and cultivator bolts are shown in the accompanying engravings, the forms illustrated being typical of a variety of bolts manufactured for agricultural apparatus.

A-Large Round Head

B — Square Head

C — Round Head, Square Shank

D - Round Head

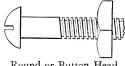
E — Key Head F — Tee Head

G — Button Head

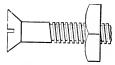
H - Concave Head

I - Reverse Key Head

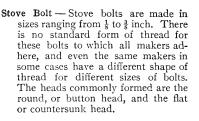
J — Large Key Head.



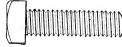
Round or Button Head



Flat or Countersunk Head







Square Head



T-Head Planer Bolt

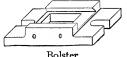
Tap Bolt — Tap bolts are usually threaded the full length of the body, which is not machined prior to running on the die. Only the point and the under side of the head are finished. They are not hardened and are used as a rule for the rougher classes of machine work. The heads are the same width as machine bolt heads.

T-Head Planer Bolt - A bolt with a T-head having oblique ends which may be dropped into the T-slot of a planer and locked by giving it a quarter turn, until the sloping ends strike against the sides of the slot. Commonly employed for holding work on the planer table.

Bolt Cutter - Machine for threading bolts, cutting threads on them.

Bolt Header - Machine for upsetting the bolt body to form the head.

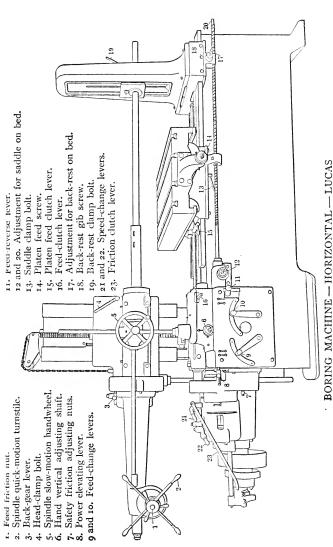
Bolster - A block sometimes called the die block, in which a punch press die is held. It is attached to the bed by bolts at either end.

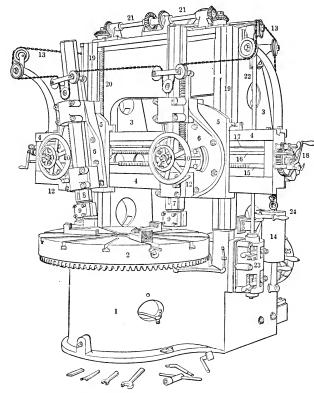


Bolster

Boring and Turning Mill - Machine having a rotating horizontal table for the work with one or more stationary vertical tools for boring, turning or facing; a turret is often provided for holding a number of tools in one of the heads.

Horizontal boring machines are not called called "vertical mill." "mills."





BORING MILL - VERTICAL - NILES

- 1. Base.
- 2. Table.
- Housing.
- 4. Cross-rail.
- 5. Saddle.
- Swivel.
- 7. Right spindle.
- 8. Left spindle.
- o. Tool heads.
- 10. Vertical feed wheels.
- 11. Power feed lock.
- 12. Spindle bearings.
- 13. Counterweight chain.

- 14. Counterweight.
- 15. Cross-feed screw.
- 16. Cross-feed screw.
- 17. Vertical feed rod.
- 18. Power feed gears.
- 10 Housing slides.
- 21. Cross-rail hoist.
- 22. Vertical power rod.
 - 23. Gear box. 24. Power control handle,

20. Vertical cross-rail screw.

- 25. Driving pulleys,
- 26. Chuck jaws.

- Box Chuck A two-jawed chuck of rectangular form used by brass finishers.
- Brass Alloy of copper and zinc although a little tin is often added for strength and density. Common proportion is copper 66, zinc 34. See bronzes, also low and high brass.
- Brass, High Only applied to rolled material. Two parts copper, one of zinc. Color is light yellow.
- Brass, Low Only applied to rolled material. Ranges from 75 per cent. copper to 25 of zinc to 88 per cent. copper and 12 of zinc.
- Brazing The joining of metals by the use of copper filings or chips and borax or some other flux. This is usually called spelter or hard solder and can be applied to almost any of the harder metals.
- Brazing Clamps Clamps to hold the ends of band saw or other work for brazing.
- Broach A tool which is practically a series of chisels or cutting edges for enlarging holes or changing their shape. Generally used for odd shaped holes but occasionally for rounds. The teeth should be on an angle to give a shearing cut. Name is sometimes given to a small reamer used by jewelers.
- Bronzes Alloy of Copper and Tin. Used in coinage, in bells, statuary, musical instruments and mirrors. Bell metal is 80 copper, 20 tin to 84 copper, 16 tin.
- Bulldozer Heavy forming machine for bending iron or steel and in which the dies move horizontally. Very similar to a forging press.
- Bull Blocks Blocks through which wire or rods are drawn to reduce size.
- Bull Wheel Usually applied to the gear of a planer which meshes into the rack under the table and drives it.



Bunsen Burner — A device for securing a very hot flame by mixing air and gas in a chamber behind the flame. The one shown has two pieces which make the flame flat instead of round.

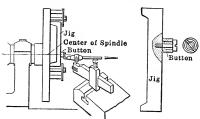
Burnishers — Tools of hardened and polished steel for finishing brass and softer metals by friction. They are held against the revolving work and give a smooth surface by compressing the outer layer of the metal.

Burring Machine - For removing burrs from hot pressed nuts.

Bushing — Tube or shell which reduces the diameter of a hole. Hardened bushings are used in jig work to guide drills or other tools.

Butt Joint — A riveted joint with the ends of the plates abutting squarely against each other.

Butt Weld — A weld in which the ends of the two pieces simply abut against each other for welding together. Button — A steel bushing, hardened and ground, used for locating a jig plate or some similar piece in which holes have to be bored



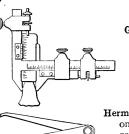
in exact position. The button is attached to the work by a small screw and is then adjusted by a micrometer or otherwise until it is central at the exact point where it is desired to bore the hole. The work is then placed on the face plate of the lathe, and with a test in-

dicator resting on the outside of the button, the piece is readily set central. It is then clamped fast to the face plate, the button is removed and the hole bored. Frequently, several buttons are used on the same piece of work, their relative positions being adjusted to conform to the center distances required between holes. The work is then indicated true by each button in succession, and one hole after another bored.

С

CALIPERS

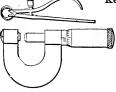
Firm Joint Calipers — Having a large, firm joint in place of old style plain riveted joint. This is an inside caliper.



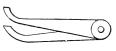
Gear Tooth Caliper — A caliper with two beams at right angles. The vertical beam gives tooth depth to pitch line and the other the thickness at pitch line. Both have verniers. Used in measuring teeth for accuracy.

Hermaphrodite Caliper — A combination of one leg of a divider and one leg of a caliper. Used in testing centered work and in laying off distances from the edge of a piece.

Keyhole Caliper — Has one straight leg and the other curved.



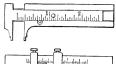
Micrometer Caliper — A measuring instrument consisting of a screw and having its barrel divided into small parts so as to measure slight degrees of rotation. Usually measure to thousandths, sometimes to ten-thousandths



Odd Leg Caliper - Calipers having both legs pointing in the same direction. Used in measuring shoulder distances on flat work, boring half round, boxes



Outside, Spring Caliper — Tool for measuring the outside diameter of work. Controlled by spring and threaded nut. Nuts are sometimes split or otherwise designed to allow rapid movement when desired, final adjustment being made by screw.



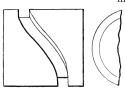
Slide Caliper — A beam caliper made with a graduated slide. Generally made small for carrying in the pocket.

Square-micrometer Caliper - A beam caliper having jaws square with the blade, and having a micrometer adjustment to read to thousandths of an inch.

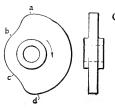


Thread Caliper - Similar to outside calipers except it has broad points to go over the tops of several threads.

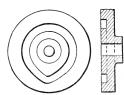
Transfer Caliper. Caliper which can be set to a given size, the auxiliary arm set, and the calipers opened at will, as they can be reset to the auxiliary arm at any time. Used to caliper recesses and places where they must be moved to get them out.



Cam, Drum or Barrel — The drum cam has a path for the roll cut around the periphery, and imparts a to-and-for motion to a slide or lever in a plane parallel to the axis of the cam. Sometimes these cams are built up of a plain drum with cam plates attached.



Cam, Edge — Edge or peripheral cams (also called disk cams) operate a mechanism in one direction only, gravity, or a spring, being relied upon to hold the cam roll in contact with the edge of the cam. On the cam shown, a to b is the drop; b to c the dwell; c to d, rise; dto a, dwell.



Cam, Face — Face cams have a groove or roll path cut in the face and operate a lever or other mechanism positively in both directions, as the roll is always guided by the sides of the slot.

"C" Clamp - See Clamp "C."

"C" Washer .- See Washer, Open.

Carbonizing — The heat treatment of steel so that the outer surface will be hard. The surface absorbs carbon from the material used.

Card Patterns — Patterns made on a gate so as to be all molded at once and to provide gates for the metal to flow.

Case-hardening — A surface hardening by which the outer skin of a piece of iron or steel absorbs carbon or carbonizes so as to harden when cooled in water. The piece is usually packed in an iron box with bone, leather or charcoal, or all three, and heated slowly several hours, then quenched.

Cat Head — A collar or sleeve which fits loosely over a shaft and is clamped to it by set screws. Used for steady rest to run on where it is not desired to run it on the work.

Same name is also given to the head carrying cutters on boring

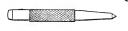
bars.

Cat Head Chuck — A chuck in which the end of a shaft or other piece is driven by a number of set screws tapped through the wall of the chuck.

Cathode — The negative terminal of an electric bath or battery.

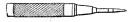
Center, Dead — The back center or the stationary center on which the work revolves. On many grinding machines both centers are dead.

Center, Live — The center in the revolving spindle of a lathe or similar machine. It is highly important that this should run true or it will cause the work to move in an eccentric path.

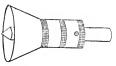


Center Punch — Punch for marking points on metal. Made of steel with a sharp point and hardened. Often called a prick punch.

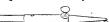
Center Punch, Automatic — Has a spring actuated hammer in the handle, which is released when the handle is pressed way down.



The point can be placed where desired and the blow given by a pressure of the hand. In some cases the blow can be varied.



point and is adjustable.



Center Punch, Bell or Self-centering - A center punch sliding in a bell or cone mouthed casing so when placed square over the end of any bar it will locate the center with sufficient accuracy for most purposes.

Center Punch, Locating - Having an extra leg which has a spring The spring point is placed in the first punch mark and so locates the next punch mark at the right distance from the first.

Centering Machines — For drilling and reaming center of work for the lathe or grinder.

Chamber — A long recess. See Recess.

Chasers - Tools used for cutting threads by chasing. Usually have several teeth of right pitch, but name is sometimes applied to a single point tool used in brass work on a Fox lathe. Chasers are made circular or flat and in the old days many were used by hand.

Chasing Threads — Cutting threads by moving a tool along the work at the right speed to give the proper pitch. Distinguishes between threads cut with a die and those cut with a threading tool.

Chattering — A slight jumping of the tool away from the work or vice-versa, and which leaves little ridges in same direction as the teeth. Occurs at times in any class of work and with any kind of tool. Due to springing of some parts of the machine.

Cherry — See Cutters, Milling.



Chisel, Blacksmith's Hot — A chisel for cutting hot metal. Has a handle so that it can be used without getting the hand too near the heated metal.

Chipping - The cutting of metal with cold chisel and hammer. Also used when a piece "chips" or breaks out of a piece or punch. Chisel, Cape - Chisel with a narrow blade for cutting keyways and similar work.



Chisel, Cold — The usual machinists' chisel for cutting or "chipping" metal with a plain cutting edge as in illustration.

Chisel, Diamond or Lozenge - Similar to a cape chisel but with square end and cutting edge at one corner. Used for cutting a sharp-bottomed groove.

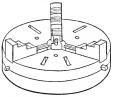
Chisel, Round — A round end chisel with the cutting edge ground back at an angle. Used for cutting oil grooves and similar work. Chuck, Draw — Operated by moving longitudinally in a taper bear-

ing. Used on precision work.

Chuck, Drill — A chuck made especially for holding drills in drilling machines. Sizes run from the smallest up to one inch.

Chuck, Eccentric — For turning eccentrics or other work in which hole is not concentric with outside. Usually made adjustable to suit varying degrees of eccentricity.

Chuck, Expanding — For turning hollow work which must be held on inside. Jaws go inside of work.



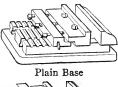
Chucks, Lathe — Devices for holding work. Usually screw on spindle and have two, three or four jaws. These may be independent or move together by screws only (in case of two jawed) or screws and gears in case of more than two. There is also a spiral or scroll chuck without gears or screws of the ordinary kind.

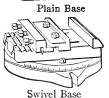
Chuck, Magnetic — Has no jaws but holds iron and steel by magnetism.

Chuck, Master — The main body of a screw chuck which screws on the nose of the lathe spindle and which carries the sub- or screwchuck for holding the work. Mostly used in brass work.

Chuck, Nipple - For holding short piece of pipe to be threaded.

Chuck, Oval — Chuck designed to move the work to and from the tool so as to produce an oval instead of a round. Sometimes called an elliptic chuck.





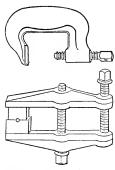
Chuck, Planer — For holding work on bed or platen of planer, shaper or milling machine. Sometimes called a vise. They are made with both plain and swivel bases as shown, and usually have locking strips which hold the piece carrying the set screws.

Chuck, Screw — Chucks made with internal or external thread to hold work which has been already threaded. These very often screw into a master chuck. Mostly used in brass work.

Chuck, Spring - See Screw Machine Tools.

Chucking Machines — Usually have a turret for tools, a revolving chuck or table for work, and generally used for boring and reaming. May be either vertical or horizontal.

Circuit — The path in which an electric current flows.



Clamp, "C" — Clamp shaped like a letter "C" for holding work in various ways. Are sometimes cast but more often drop forged for heavier work.

Clamps, Machinist — Clamps for holding work together, holding jigs or templets on work, etc.

Clash Gears - Gears which are thrown into mesh by moving the centers together and sometimes by sliding the gears on parallel shafts till the teeth get a full bearing. The latter are sometimes called sliding gears.

Clutch - Any device which permits one shaft to engage and drive another, may be either friction or positive, usually the former. Made of all sorts of combinations of cams, levers and toggles.

Clutch, Friction - A device whereby motion of loose pulley is transmitted to shaft to be driven. Various methods are employed but all depend on forcing some kind of friction surfaces together so that one drives the other.



machines to a constantly running shaft

or one part to another, at will. are many kinds, both positive and friction. The illustrations show two of the most common of the positive clutches.

Clutches, Positive - Devices for connecting

Square Taw Clutch

Collar - A ring used for holding shafting, loose pulleys, in proper position or for fastening to boring tools to prevent them going in too deep.

Collar, Safety - Having a clamping device instead of set screw or having set screw below surface or so covered as not to catch anything brought in contact with it.

Commutator - The part of a dynamo or motor which takes off or leads current into the machine.

Compound Rest - An auxiliary tool slide on lathe carriage arranged to swivel so as to turn at any desired angle with the lathe centers or with cross slide. Usually graduated into degrees.

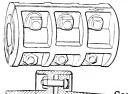


Cotter, Spring — Also called split cotter, split pin, etc., is used in a hole drilled crosswise of a stud, shaft or some similar member, and its ends spread apart to retain it in place and keep some member carried by the shaft from slipping off.

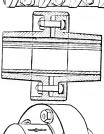


Counterbore — Has a pilot to fit a hole already drilled, or drilled and reamed, and its body with cutting edges on the end is used to enlarge the hole to receive a screw head or body or for some similar purpose.

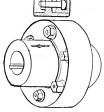
Countershaft — The shaft for driving a machine which is itself driven by the main or line shaft.



Coupling, Clamp - Couplings made in two or more parts, clamping around the shafts by transverse bolts. Hold either by friction or have dowels in shaft. Sometimes called compression although this is confusing.



Coupling, Compression — Grips shafting by drawing together tapered parts. forces them against shaft and holds it Bolts parallel with shaft draw parts together.



Coupling, Flanged — A flange is keyed to each shaft and these flanges are bolted together. Also called "plate" coupling.

Coupling, Friction - Couplings which depend on frictional contact. Coupling, Jaw or Clutch - Positively engaged by jaws or projections on the face of opposing parts.

Coupling, Shaft — Devices for fastening ends of shafting together so that both may be driven as one shaft. These are made in a great variety of ways, from plain set screw coupling to elaborate compression devices.

Coupling, Wedge — Coupling that clamps the shaft with a wedging action. Practically like a compression coupling. Generally enclosed in a sleeve. Also called vise coupling.

Cope — The upper part of a flask.

Coping Machine — For cutting away the flanges and corners of beams and bending the ends.



Counter, Revolution — Device for counting the revolutions of a shaft. Generally made with a worm and a gear having 100 teeth so that one turn of dial equals 100 revolutions.

Countershaft — Shaft carrying tight and loose pulleys (or friction clutch pulleys) for starting and stopping machines or reversing their motion.

Crane, Gantry — Traveling crane mounted on posts or legs for yard use.

Crane, Jib - Crane with a swinging boom or arm.

Crane, Locomotive — Crane mounted on a car with an engine so as to be self-propelling on a track.

Crane, Pillar — Having the boom or moving arm fastened to pillar or post.

Crane, Portable — Hoisting frame on wheels which can be run around to the work and used to handle work in and out of lathes and other machines.

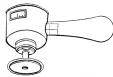
Crane, Post - See Crane, Pillar.

Crane, Swing - Same as Jib Crane.

Crane, Traveling — Crane with a bridge or cross beam having wheels at each end so it can be run on overhead tracks to any point in the shop.

Crimping — Fluting, corrugating or compressing metal ring to reduce its diameter.

Cross-rail — The part of a planer, boring mill or similar machine on which the tool heads or slides move and are supported.



Cut Meter — Instrument for measuring the surface speed of work either in lathe or planer. A wheel is pressed against the moving surface and the speed is shown in feet per minute.

Cutters, Flue Sheet — Special cutters for making holes as for flues, in flue sheets or in other sheet metal or structural work.

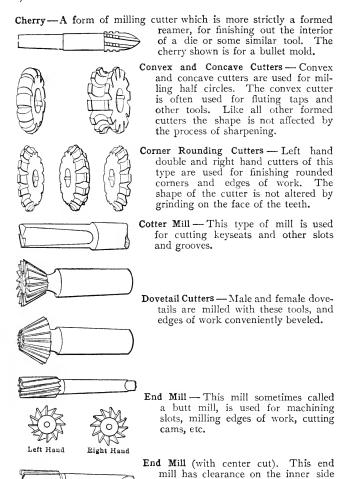
CUTTERS, MILLING





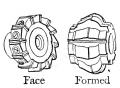


Angular Cutters — Such cutters are used for milling straight and spiral mills, ratchet teeth, etc. Cutters for spiral mill grooving are commonly made with an angle of 12 degrees on one side and 40, 48 or 53 degree angle on the other.



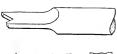
of the end teeth and is adapted to Left Hand Mill cut into the work to a depth equal to the length of the end teeth and then feed along, dispensing with the necessity of first drilling a hole, which has to be done when the inner sides Left Hand Right Hand of the teeth are not relieved.

The mills are often used for heavy cuts particularly in cast iron.

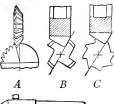


Face and Formed Cutters — The face cutter to the left, of Brown & Sharpe inserted tooth type is made in large sizes and cuts on the periphery and ends of teeth. The formed cutter to the right may

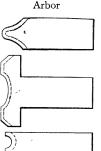
The formed cutter to the right may be sharpened by grinding on the face without changing the shape. For milling wide forms several cutters are often placed side by side in a gang.



Fish Tail Cutter — A simple cutter for milling a seat or groove in a shaft or other piece. Usually operated at rapid speed and light cut and feed.



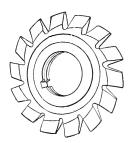
Fluting Cutters — Cutter A is an angular mill for cutting the teeth in spiral mills, cutter B is for tap fluting and C for milling reamer flutes. In each case the cutter is shown with one face set radial to the center of the work.



Fly Cutters — Fly cutters are simple formed cutters which may be held in an arbor like that shown at the top of the group. The arbor is placed in the miller spindle and the tool or other work to be formed is given a slow feed past the revolving cutter. After roughing out, the cutter can be held stationary and used like a planer tool for finishing the work which is fed past it and so given a scraping cut.



Gang Cutters — Cutters are used in a gang on an arbor for milling a broad surface of any desired form. The cutters shown have interlocking and overlapping teeth so that proper spacing may be maintained. In extensive manufacturing operation the gangs of cutters are usually kept set up on their arbor and never removed except for grinding.



Gear Cutter (Involute). In the Brown & Sharpe system of involute gear cutters, eight cutters are regularly made for each pitch, as follows:

No. 1 will cut wheels from 135 teeth to a rack.

No. 2 will cut wheels from 55 teeth to 134 teeth. No. 3 will cut wheels from 35 teeth

to 54 teeth. No. 4 will cut wheels from 26 teeth

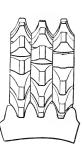
to 34 teeth.

No. 5 will cut wheels from 21 teeth to 25 teeth.

No. 6 will cut wheels from 17 teeth to 20 teeth.

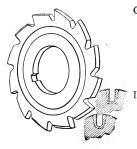
No. 7 will cut wheels from 14 teeth to 16 teeth. No. 8 will cut wheels from 12 teeth to 13 teeth.

Such cutters are always accurately formed and can be sharpened without affecting the shape of the teeth.



Gear Cutters, Duplex — The Gould & Eberhart duplex cutters are used in gangs of two or more; the number of cutters in the gang depending on the number of teeth in the gear to be cut. The following table gives the number of cutters which may be used in

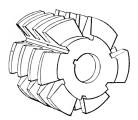
cutting different numbers of teeth. Under 30 teeth 1 cutter Over 30 teeth 2 cutters Over 50 teeth 3 cutters Over 70 teeth 4 cutters Over 95 teeth 5 cutters Over 120 teeth 6 cutters Over 150 teeth 7 cutters Over 180 teeth 8 cutters Over 230 teeth 10 cutters Over



Gear Stocking Cutter - The object of stocking cutters is to rough out the teeth in gears, leaving a smaller amount of metal to be removed by the finishing cutter. They increase the accuracy with which gears may be cut, and save the finishing cutter as well.

260 teeth 12 cutters.

In all cases where accuracy and smooth running are necessary the gears should first be roughed out. One stocking cutter answers for all gears of the same pitch.



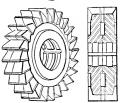
Hob - A form of milling cutter with teeth spirally arranged like a thread on a screw and with flutes to give cutting edges as indicated. Used for cutting the teeth of worm gears to suit the worm which is to operate the gear. Hobs are formed and backed off so that the faces of the teeth may be ground without changing the shape.



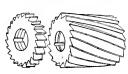
Inserted Tooth Cutter — Brown & Sharpe inserted tooth cutters have taper bushings and screws for holding the blades in position in the bodies. Inserted tooth construction is generally recommended for cutters 6 inches or larger in diameter. There are many types of inserted tooth cutters and in most cases the blades are readily removed and replaced when broken or worn out.



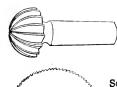
Inserted Tooth Cutter (Pratt & Whitney). this type of cutter the teeth or blades are secured in position by taper pins driven into holes between every other pair of blades; the cutter head being slotted as shown to allow the metal at each side of the taper pin to be pressed firmly against the inserted blades.



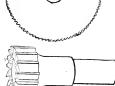
Interlocking Side Cutters - These cutters have overlapping teeth and may be adjusted apart to maintain a definite width for milling slots, etc., by using packing between the inner faces.



Plain Cutters — These cutters are for milling flat surfaces. When over 3 inch wide the teeth are usually cut spirally at an angle of from 10 to 15 degrees, to give an easy shearing When of considerable length relative to diameter they are called slabbing mills.



Rose Cutter — The hemispherical cutter known as a rose mill is one of a large variety of forms employed for working out dies and other parts in the profiler. Cutters of this form are also used for making spherical seats for ball joints, etc.

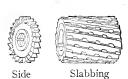


Screw Slotting Cutter — Screw slotting cutters have fine pitch teeth especially adapted for the slotting of screw heads and similar work. The cutters are not ground on the sides. They are made of various thicknesses corresponding to the numbers of the American Wire Gage.

Shank Cutter — Shank milling cutters are made in all sorts of forms with shanks which can be conveniently held true in miller or profiler while in operation.



Shell End Cutter — Shell end mills are designed to do heavier work than that for which the regular type of end mills are suited. They are made to be used on an arbor and are secured by a screw in the end of the arbor. The end of the cutter is counterbored to receive the head of the screw and the back end is slotted for driving as indicated.



Side or Straddle, and Slabbing Cutters—
Side cutters like that to the left cut on the periphery and sides, are suitable for milling slots and when used in pairs are called straddle mills. May be packed out to mill any desired width of slot or opposite faces of a piece of

any thickness.

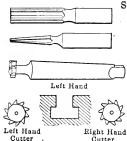
Slabbing cutters are frequently made with nicked teeth to break up the chip and so give an easier cut than would be possible with a plain tooth.



Slitting Saw, Metal — Metal slitting saws are thin milling cutters. The sides are finished true by grinding, and a little thicker at the outside edge than near the center, for proper clearance. Coarse teeth are best adapted for brass work and deep slots and fine teeth for cutting thin metal.

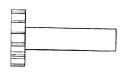


Sprocket Wheel Cutter — Cutters for milling the teeth on sprocket wheels for chains are formed to the necessary outline and admit of grinding on the face like regular gear cutters, without changing the form of the tooth.



Straight Shank Cutter — Straight shank cutters of small size are extensively used in profilers and vertical millers for die sinking, profiling, routing, etc. They are held in spring chucks or collets.

T-Slot Cutter — Slots for bolts in miller and other tables are milled with T-slot cutters. They are made to standard dimensions to suit bolts of various sizes. The narrow part of the slot is first milled in the casting, then the bottom portion is widened out with the T-slot cutter.



Woodruff Key Cutter — The Whitney Mfg, Co's keys are semi-circular in form and for cutting the seats in the shaft to receive them a cutter of the type shown is used. These cutters are made of right diameter and thickness to suit all the different sizes of keys in the Woodruff system.

Cutting-off Machine — For cutting desired lengths from commercial bars of iron, steel or other material, usually has stationary tools and revolves the work. The latter is gripped by the rotating chuck; and as the tools are fed toward the center, the spindle in some types of machines is driven at an accelerated speed so that as the diameter of the cut is reduced, the speed of rotation is increased to maintain a practically uniform surface speed of work. In cold-saw cutting-off machines, the work is held in a vise and a rotating circular cutter is fed against it. Such machines are used not only for severing round stock but also for cutting off square and rectangular bars, rails, I-beams, channels and other structural steel shapes.

Cutting-off Tool Post — The tool block used on the cross slide of a turret lathe or other machine for carrying the tool for cutting off the completed piece of work from the bar of material held in the chuck. The tool post may be made to receive a straight tool or a circular cutter.

D

Daniels Planer - See Planer.

Dead Center - The center in the tail spindle of a lathe or grinder which does not revolve.

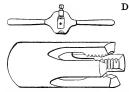
Derrick — Structure consisting of a fixed upright and an arm hinged at bottom, which is raised and lowered and usually swings around to handle heavy loads.

Dial Feed — A revolving disk which carries blanks between the punch and die.



Diamond Hand Tool — Used for dressing grinding wheels after they have been roughed out with the cheaper forms of cutters. Fixed diamonds are usually considered better than those held by hand.

Die Chasers - Threaded sections inserted in a die head for cutting bolts and screws.



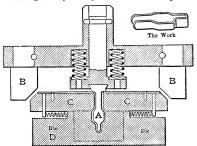
Die, Screw Plate or Stock - A frame or handle for holding a threading die. Sometimes die and handles are of one piece.

Die, Spring or Prong - Die with cutting portions in the end of prongs; can be adjusted somewhat by springing prongs together with a collar on outside.

Dies, Bolt - Dies for cutting bolts. Some are solid, others adjustable. Some for hand die stocks or plates but mostly for machine bolt cutters.

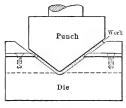
DIES, PUNCH PRESS

Bending Dies, Compound - In compound bending dies of the type



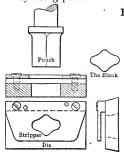
shown the work is carried down into the die by punch A, and held there while the beveled fingers B act upon the slides C and cause them to move forward in the top of die D and bend the material to the outline of the punch. Upon the up-stroke of the punch the slides C are pressed outwardly by their springs and the

bent piece of work is removed by the punch from the die. It will be seen that the holder for punch A, upon which depends the interior form of the piece being bent, is not positively secured in its holder, but is instead adapted to slide up and down in its seat, although prevented from turning by a small pin at the upper end of the shank which is engaged by a slot in the punch carrier. The springs shown above the punch proper tend to hold the punch in its lower position and at the same time after the punch has passed down into the die allow the punch carrier to descend still further to press fingers B into operation against slides C which bend the work to the outline of the punch.

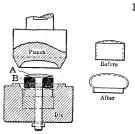


Bending Dies, Plain — Simple bending dies are made with the upper face of the die and the bottom of the punch shaped to conform to the bend it is desired to give the blank. A common type is shown in the engraving.

In simple bending dies the upper face of the die is cut out to the desired form and the piece of work formed to required shape by being pressed directly down into the die by the punch.



Blanking Dies — Blanking dies are about the most commonly used of all the varieties of press tools. A simple form of die is seen in the illustration. The strip of sheet metal is fed under the stripper and is prevented by that member from lifting with the punch upon the up-stroke, following the punching out of the blank. Where several punches are combined in one hole for blanking as many pieces simultaneously, they are known as multiple blanking tools.

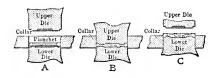


Bulging Dies — The "before and after" sketches show the character of the work handled in bulging dies. The shell after being drawn up straight is placed over mushroom plunger A in the bulging die, and when the punch descends the rubber disk B is forced out, expanding the shell into the curved chamber formed by the punch and die. Upon the punch ascending, the rubber returns to its original form and the expanded work is then removed.

Burnishing Dies — Burnishing dies are made a little smaller at the bottom than at the top and when the work is forced down through the die, the edge of the piece is given a very high finish making

polishing and hand finishing operations unnecessary. The burnishing process forms a very accurate sizing method also.

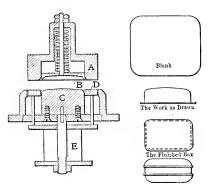
Coining Dies - Coining dies are operated in very powerful presses



of the embossing type similar to those used for forming designs on silverware, medals, jewelry, etc. The position of the work, the retaining coller and the dies are indicated at A, B, and C.

In the latter section the coin is shown delivered at the top of the die.

Combination Dies — Combination dies are used in single acting presses for such work as cutting a blank and at the same stroke



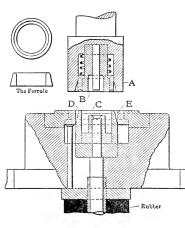
turning down the edge and drawing the piece into the required shape. In most cases the work is pushed out of the dies by the action of a spring. Such a set of dies is shown in the engraving, for making a box cover and body. The work is blanked by cuttingpunch A, and formed to the right shape by B and C, the former holding the piece by spring pressure to the block C while punch A, continuing to descend,

draws the box to the required shape. Ring D acts as an ejector or shedder and is pressed down, compressing the rubber at E during the drawing operation, and upon the up-stroke of the punch, ascends and ejects the work from the dies.

Dies of this type, with a spring actuated punch or die inside the regular blanking tool, are often used for simultaneously blanking

and piercing, blanking and bending, etc.

It will be noticed that the lower view in the group showing the work at the right-hand side of the sectional illustration of the die, represents the box cover and body as they appear when assembled after the superfluous metal in the flange or "fin" has been removed in a trimming die. This fin as left on the piece when coming from the combination die is shown in the view immediately under the blank. A trimming die for finishing such work evenly is shown on page 507.

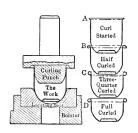


Compound Dies - Com-

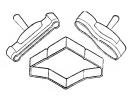
pound dies have a die the upper punch a punch in lower die. The ferrulemaking tools shown have a blanking and outer drawing punch A, with a central die B, to relower punch C which cuts out the center of the ferrule blank and allows the metal to be drawn down inside as well as outside of the bevel edged member D. As the work is drawn down ring E descends compressing rubber cushion F below and upon the return movement the ferrule is ejected from the die.

Cupping Dies - Used for drawing up a cup from a disk or planchet.

Same as drawing dies.

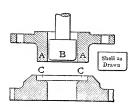


Curling Dies - Curling dies are used for producing a curled edge around the top of a piece drawn up from sheet metal. When the top is to be stiffened by a wire ring around which the metal is curled, a wiring die is used, the construction of which is practically the same as the curling die. The illustration shows a curling die and the appearance of a shell various stages during the operation of curling over the edge. The diagrams A, B, C, D, show the progress in the curling process as the punch descends, pressing down on the edge of the straight drawn shell.



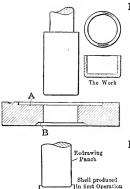
Dinking Dies - Dinking dies, or hollow cutters, although not usually classed with regular dies are used so commonly as to entitle them to be listed in that class. They are adapted to punching out all sorts of shapes from leather, cloth, or paper. edges of the dies (a few specimens of which are shown in the engraving) are usually beveled about 20 degrees out-Where made for press use the handle is omitted. As a surface for the cutting edge of the die to strike on, a block is built up of seasoned rock maple, set endwise of the grain.

Double Action Dies — This type of die is used in a press which has a double acting ram; that is, there are two slides, one inside the



other, which have different strokes. To the outer slide is fastened the combined cutting punch and blank holder A, which is operated slightly in advance of the drawing punch B actuated by the inner slide. The blank upon being cut from the stock by A, drops into the top of the die at C and is kept under pressure by the flat end of cutting punch A to prevent its wrinkling, while punch B continues downward drawing the metal from

between the pressure surfaces and into the shape required.

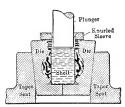


Drawing Dies, Plain — Dies of the type shown can be used for shallow drawing only, as there is no pressure on the blank to prevent its wrinkling when forced down into the die by the punch. The blank fits the recess A in the upper face of the die and the die itself which is slightly tapering is made the diameter of the punch plus twice the thickness of the wall required for the shell. The bottom edge B of the die strips the shell from the punch when the latter ascends.

Re-drawing dies are used for drawing out a shell or cup already formed from the sheet metal. In the illustration, a shell ready for re-drawing is shown in position in the dies, which need little explanation. The work is located in the upper plate A, and after being forced through the die B, is stripped from the punch by edge C. Ordinarily, a shell which is to be

given considerable elongation, is passed through a number of redrawing dies.

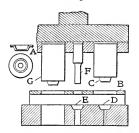
Re-drawing dies are sometimes referred to as reducing dies, although the latter, as explained under the proper heading on page 504, are used for drawing down the end of a shell only, as in the case of a cartridge shell, which is made with a neck somewhat smaller in diameter than the body.



Fluid Dies — Water or fluid dies are used

for forming artistic hollow ware of silver, and other soft metals, in exact reproduction of chased work. The die as shown is a hinged mold cast from carved models and finished with all details clean and sharp. The shell to be worked is filled with liquid and enclosed in the die and a plunger in the press ram then descends and causes the fluid to force the metal out into the design in the die.

Follow Dies — Follow tools consist of two or more punches and dies in one punch holder and die body, these being arranged in

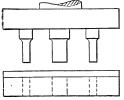


and die body, these being arranged in tandem fashion so that after the first operation the stock is fed to the next point and a second operation performed; and so on. In the die shown, which is for making piece A, the strip of metal is first entered beneath stripper. B far enough to allow the first shell to be drawn at the first stroke by punch C and die D. The metal is then moved along one space and the shell drawn at the first stroke is centered and located within the locating portion of piercing die E. At the

in a blank at one stroke of the press. Sometimes dies which perform a number of operations on a piece which is fed along successively under one punch after another are called "gang" dies; strictly speaking, however, such tools

next stroke the hole in the center is pierced with punch F, and a second shell drawn in the stock by punch and die C and D. The stock is then fed forward another space and the blanking punch G cuts out the piece from the metal. At the same stroke a third piece is being formed on the end of the stock and a second hole pierced. Thus three operations are carried on simultaneously.

Gang Dies — Gang tools have two or more punches and dies in one holder for making as many openings

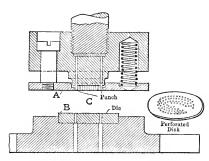


are "follow" dies. Where a large number of punches are combined they are called a multiple punch, or if they are perforating punches.

Heading Dies — Heading dies strike up the heads on cartridges and other shells, and are generally operated in a horizontal heading machine.

Index Dies — For certain classes of work such as notching the edges of large disks or armature punchings, an index die is sometimes used consisting of a rotary index plate adapted to carry the work step by step past the punches which cut out one notch or a series of notches at each stroke of the press.

Perforating Dies -- Perforating tools consist of a number of piercing



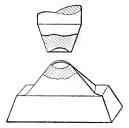
tools in one set of dies and may be called also multiple piercing tools. In the example shown, which punches a large number of holes in a disk, the work is held by the spring-controlled pressure-pad A against the face of the die B while the punches at C are forced down through the sheet metal. In this case the punches are easily replaced when broken, by unscrewing

the holder from the shank and slipping the small punch out from the back.

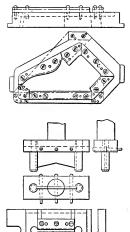
Piercing Dies — Piercing tools are used for punching small holes through sheet metal. Where arranged for punching a large number of holes simultaneously they are often called perforating dies.

Piercing Dies, Compound — Compound piercing tools have, in addition to the regular punches carried by the holder in the ram, a set of horizontal punches for making holes through the sides of the work. These side punches are operated by slides moved inward by wedge-shaped fingers, the arrangement being the same as in the case of the compound bending dies, an illustration of which is given under that head.

Reducing Dies — Reducing dies are re-drawing dies for reducing a portion of the shell only, whereas the regular re-drawing die reduces the whole length. Reducing dies for cartridge shells form the familiar "bottle neck" shell now so commonly manufactured, with a larg body for the powder and a smaller neck into which the bullet is secured. In dial feed presses ordinarly employed for cartridge making operations, two or more reducing dies are often used for shaping the neck of the shell to the required dimensions, each die operating in turn upon the shell as it is carried around step by step under the press tools by the intermittent rotary movement of the feeding dial.



Riveting Dies — Riveting dies for the punch press are provided with cavities in the working faces to suit the shape of the head it is desired to produce on the ends of the rivets.

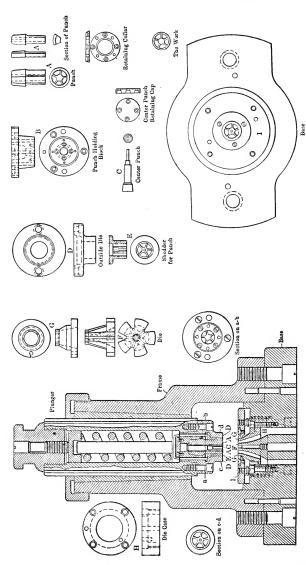


Sectional Dies — Frequently dies of complicated outline are built up in sections to enable them to be more easily constructed and kept in order. This form is resorted to often in the case of large dies where a break at one point would mean considerable expense for a new die. Also the difficulties of hardening are reduced with the sectional construction. As shown, the various parts are secured to a common base or holder.

Shearing Dies — Shearing dies are used for cutting-off operations, and are frequently combined with other press tools so that after certain operations on a piece it can be severed from the end of the stock. The shearing tools in the engraving are arranged for simply cutting up stock into pieces of the required length and the punch itself is of the inserted type secured by pins in its holder.

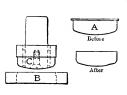
Split Dies — Split dies form one type of sectional die — the simplest; they are made in halves to facilitate working out to shape, hardening, and economical maintainance.

Sub-press Dies — A sub-press and its tools are represented on the following page. Such tools are used for small parts which have to be made accurately and are very common in watch and type-writer shops and similar places. The tools are held positively in line in this press and as a result their efficiency is maintained indefinitely. The press is slipped bodily into the regular power press with the base clamped to the press bed and the neck of the sub-press plunger connected with the ram of the press.



Swaging Dies — Swaging operations are resorted to where it is desired to shape up or round over the edges of work already blanked out. Thus in watch wheel work the arms and inside edges of the rims are sometimes swaged to a nicely rounded form subsequently to the blanking out of the wheel in the sub-press. Swaging dies for such work are of course made with shallow impressions which correspond to a split mold between the two halves of which the blank is properly shaped.

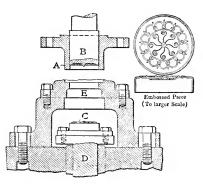
Bullet swaging dies receive the slug as it comes from the bullet mold and shape the end to the required cone point.



Trimming Dies — Trimming dies remove the superfluous metal left around the edges or ends of various classes of drawn and formed work. In the case shown, the box body A has been drawn up and a fin left all the way round; this is dropped into the trimming die B and the punch C in carrying it through the die trims the edge off evenly, as indicated. Work of the nature

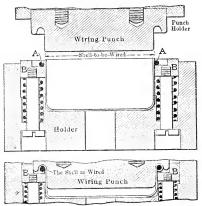
shown in this illustration is blanked, drawn up and formed ready for trimming, by means of combination tools, a typical example of which will be found under the combination die heading on page 500. The box body illustrated here as it appears before and after trimming is shown in connection with the combination dies as it appears in the blank, after it is formed, and after assembling with its cover.

Triple-action Dies - These dies are used in triple-acting presses,



where in addition to the double-action slides which take the place of the regular single-acting there is also a third slide or plunger which operates under the table or die bed. Thus a piece like that. shown which has to be blanked, drawn embossed, is operated upon from above by the cutting and drawing punches A and B, upon the latter carrying the drawn work down to the face

of the embossing die C, that die is forced upward by the plunger D beneath and gives the piece the desired impression. On the up-stroke of the punch the work is stripped from it by edge E and falls out of the press.



Wiring Dies - Wiring dies are much the same in construction as plain curling dies. In the engraving, the wire ring is shown at A around the top of the shell to be wired and in a channel at the top of the springsupported ring B. As indicated in the lower illustration, the punch as it descends, depresses the ring Band curls the edge of the shell around the wire ring A.

Disks, Reference — Accurate disks of standard dimensions for setting calipers and measuring with. Usually of hardened steel.



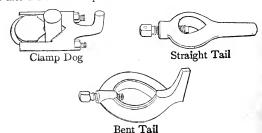
Divider, Spring — The spring tends to force the points apart and adjustments are made by the nurled nut on the screw.

Doctor — Local term for adjuster or adapter so that chucks from one lathe can be used on another. Sometimes used same as "dutchman."

Dog — Name given to any projecting piece which strikes and moves some other part, as the reversing dogs or stops on a planer or milling machine. Sometimes applied to the pawl of a ratchet.

Dog, Clamp — Grips work by clamping with the two parts of the dog. There are many types both home made and for sale.

Dog, Lathe — Devices for clamping on work so that it can be revolved by face-plate. Straight tail dogs are driven by a stud on face-plate. Curved tail (usual way) dogs have the end bent to go into a slot in face-plate.



Drag — The bottom part of a flask, sometimes called the nowell.

Draw Bench — Place where wire is drawn from rods, being drawn through plates or bull blocks with successively smaller openings.

Drift — A tool for cutting out the sides of an opening while driven through with a hammer.

DRILLS



Center Drill — The short drills used for centering shafts before facing and turning are called center drills. The drill and reamer or countersink for the 60 degree center hole when combined as shown allow the centering to be done more readily than when separate tools are used.

Core Drill — The core drill is a hollow tool which cuts out a core instead of removing the metal in the form of chips. Such drills are generally used to procure a

core from the center of castings or forgings for the determination of the

tensile strength or other physical properties of the metal.

Gun Barrel Drill — Gun barrel drills are run at high speed and under very light feed, oil being forced through a hole in the drill to clear the chips and cool the cut-

ting point and work. The drill itself is short and fastened to a shank of suitable length.

Hog Nose Drill — More like a boring tool. Mostly used for boring out cored holes. Must be very stiff to be effective but when made right and used to advantage, does lots of hard work.

Hollow Drill — The hollow drill is for deep-hole drilling. It has an opening through the body and is

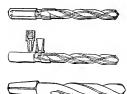
side as shown.



attached to a shank of the necessary length for the depth of hole to be drilled.

Oil-drill (Morse) - These

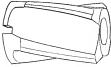
drills convey lubricant to the point, through holes formed in the solid metal. Where the drills are larger than 2½ inches an inserted copper tube is employed to carry the oil to the drill point and wash out the chips and keep the drill cool. The oil enters through the hollow shank or through a connection at the



Twist

Ratchet Drill — The square taper shanks of these drills are made to fit a ratchet for drilling holes by hand.

Flat

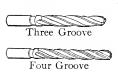


Shell Drill — Shell drills are fitted to a taper shank and used for chucking out cored holes and enlarging holes drilled with a two-flute twist drill. The angle of the spiral lips is about 15 degrees.

Straight Flute Drill — The straight flute, or "Farmer" drill as it is frequently called after its inventor, does not clear itself as well as the twist drill does but is stiffer

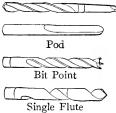
as the twist drill does, but is stiffer, and does not "run" or follow blowholes or soft spots as readily as the

twist drill. It is also better for drilling brass and other soft metals.



Three and Four-groove Drills — Where large holes are to be made in solid stock, it is advisable to use a three or four groove drill after running the required two-flute drill through the piece. These drills will enlarge the hole to the size required and are also useful in boring out cored holes in castings.

Twist Drill — Usually made with two flutes or grooves, running around the body. This furnishes cutting edges and the chips follow the flutes out of the hole being drilled.



Wood Drill (Bit) — Bits for wood drilling are made in various forms. The pod drill is cut out hollow at the working end; the double flute spiral drill has a regular bit point; the single flute drill is full diameter for a short distance only and is cleared the rest of the length as indicated.

Drill, Chain — Device to be used in connection with a brace or breast drill in many places where it is not convenient to bring a ratchet drill into use.

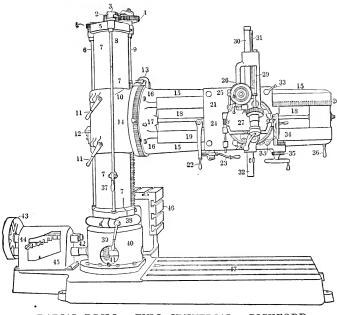
Drill Speeder — Device which goes on drill spindle and gears up the speed of drills so that small drills can be used economically on large drill presses.

Drill Vise — See Vise, Drill.

Drill, Radial - Parts of

- 1. Vertical driving-shaft gear.
- Center driving-shaft gear.
 Elevating tumble-plate seg-
- ment.
 4. Elevating-screw gear.
- 5. Column cap.

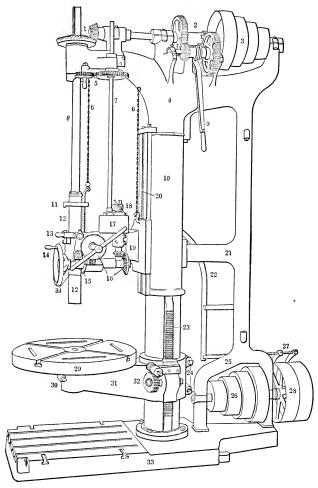
- 6. Vertical driving shaft.
- 7. Column sleeve.
- 8. Elevating-lever shaft.
- 9. Elevating screw.
- 10. Arm girdle.
- 11. Arm-binder handle.



RADIAL DRILL—FULL UNIVERSAL—BICKFORD

- 12. Arm-miter gear guard.
- 13. Arm-worm box.
- 14. Arm pointer.
- 15. Full universal arm.
- 16. Arm-clamping nuts.
- 17. Arm-dowel pin.
- 18. Arm shaft.
- 10. Arm ways.
- 20. Arm rack.
- 21. Saddle.
- 22. Reversing lever.
- Back-gear lever.
- 24. Head-swiveling worm.
- 25. Feed-trip lever.
- Index gear.
- 27. Universal head.
- 28. Quick-return lever.
- 29. Feed-rack worm shaft.

- 30. Spindle sleeve.
- 31. Feed rack.
- 32. Spindle.
- 33. Saddle-binding lever.
- 34. Feed hand wheel.
- 35. Head-moving gear.
- 36. Arm-swinging handle.
- 37. Elevating lever.
- 38. Clamping ring.
- 30. Clamping-ring handle.
- 40. Column.
- 41. Column driving-miters.
- 42. Driving-shaft coupling.
- 43. Driving pulley.
- 44. Speed-change lever.
- 45. Speed-box case,
- 46. Box table.
- 47. Base.



DRILL PRESS — CINCINNATI MACHINE TOOL COMPANY

Drill Press - Parts of

- 1. Main driving gears, bevel.
- 2. Back gears.
- 3. Upper cone pulley.
- 4. Yoke to frame.
- 5. Feed gears.
- 6. Counterweight chains.
- 7. Feed shaft.
- Spindle.
- 9. Back-gear lever
- 10. Column
- II. Automatic stop.
- 12. Spindle sleeve.
- Feed-trip lever.
 Hand-feed wheel.
- 15. Quick-return lever.
- 16. Feed gearing.
- 17. Feed box.

- Dia Press Parts of
 - 18. Feed-change handle.
 - 10. Sliding head.
 - 20. Face of column.
 - 21. Back brace. 22. Belt shifter.
 - 23. Rack for elevating table.
 - 24. Table-arm clamping screws.
 - 25. Pulley stand.
 - 26. Lower cone pulley.27. Belt-shifting fingers.
 - 28. Tight and loose pulleys
 - 29. Table.
 - 30. Table-clamp screw.
 - 31. Table arm.
 - 32. Table-adjusting gear.
 - 33. Base.
 - 34. Ball-thrust bearing.

Drive or Force Fit - See Fit.

Dry Sand Molds — Molds made of green sand and baked dry in ovens or otherwise dried out before pouring.

Dutchman — Local term for a wedge or liner to make a piece fit. Used to make a poor job useable. A round key or pin fitting endwise in a hole drilled half in a shaft and half in the piece to be attached thereto.

Ε

Ejector — An ejector on punch press work is a ring, collar or disk actuated by spring pressure or by pressure of a rubber disk, to remove blanks from the interior of compound and other dies. It is often called a shedder.

Elliptic Chuck — See Chuck, Oval.

Emery Jointer — Grinder for making a close joint between the share and mold board of steel plows.

Emery Wheel Dressers — See Grinding Wheels and Diamonds.

End Measuring Rod — Arranged for internal measurements similar to the internal cylindrical gages.

Expanding Arbor or Mandrel — See Arbor.

Extractor, Oil — Machine for extracting oil from iron and metal chips. Revolves rapidly and throws out the oil by centrifugal force.

F

Face Cam — See Cam, Face.

Face Plate — The plate or disk which screws on the nose of a lathe spindle and drives or carries work to be turned or bored. Sometimes applied to table of vertical boring mill.

Face Reamer - See Reamer, Face.

Feather — Might be called a sliding key — sometimes called a spline. Used to prevent a pulley, gear or other part, from turning on the shaft but allows it to move lengthwise as in the feed shaft used on most lathes and other tools. Feather is nearly always fastened to the sliding piece.

Field - Usually the stationary part of a dynamo or motor.

Files — Tools of hardened steel having sharp cutting points or teeth across their surface. These are forced up by a chisel and hammer.

Filing Machine — Runs a file by power, usually vertically. Useful in many kinds of small work.

Fin — The thin edge or mark left by the parting of a mold or die.

In drop forge work this is called the "flash."

Fit, Drive or Force or Press — Fitting a shaft to a hole by making the hole so the shaft can be driven or forced in with a sledge or some power press, often requiring many tons pressure.

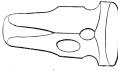
Fit, Running or Sliding — Enough allowance between shaft and hole to allow it to run or slide without sticking or heating.

Fit, Shrink — Fitting a shaft to a hole by making the hole slightly smaller than the shaft, then heating the piece with the hole till it expands enough to allow shaft to enter. When cool the shaft is very tightly seized if the allowance is right.

Fit, Wringing — A smaller allowance than for running but so that the shaft can be twisted into the hole by hand. Usually applied to some such work as a boring bar in a horizontal boring machine. Sometimes used in connection with twisting two flat surfaces together to exclude the air.

Flask — The frame which holds the sand mold for the casting. Includes both the cope and drag.

Flat Reamer — See Reamer, Flat.



Flatter — Round face. A blacksmith's tool which is held on the work and struck by a sledge. Used to take out hammer marks and smooth up a forging.

Flute — Shop name for a groove. Applied to taps, reamers, drills and other tools.

Fly-wheel — Heavy wheel for steadying motion of machinery. On an engine it carries the crank past the center and produces a uniform rotation.

Follower Rest — A back rest for supporting long lathe work; attached to the carriage and following immediately behind the turning tool.

Foot Stock — The tail stock or tail block of a lathe, grinder, etc.

Force — A master punch which is used under a powerful press to form an impression in a die. Forces are commonly employed in the making of coining and other embossing dies. A similar tool used by jewelers is called a "hub." It is sometimes referred to incorrectly as a "hob."

forge - Open fireplace for heating metals for welding, forging, etc. Has forced draft by fan or bellows.

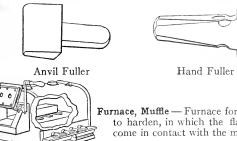
forging Press - Heavy machine for shaping metal by forcing into dies by a steady pressure instead of a sudden blow as in drop forging. Similar to a bulldozer.

Fork Center - A center for driving woodwork in the lathe. Also used in hand or fox lathes for driving special work.

fox Lathe — Lathe for brass workers having a "chasing bar" for cutting threads and often has a turret on the tail stock.

Franklin Metal - An alloy having zinc as a base, used for casting in metal molds.

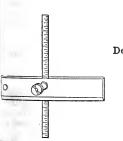
Fuller - Blacksmith's tool something like a hammer, having a round nose for spreading or fulling the iron under hammer blowers.



Furnace, Muffle — Furnace for heating steel to harden, in which the flame does not come in contact with the metals.

Furniture — In machine shops applies to tool racks, lathe pans, tote boxes, etc.

Fuse — A piece of metal which melts when too much current passes and acts as a safety valve.



GAGES

Depth Gage - A tool for measuring the depth of holes or recesses. The body is placed across the hole while the rule is slipped down into the hole to be measured. In many cases the rod is simply a wire and not graduated.

Drill Gage - Flat steel plate drilled with different size drills and each hole marked with correct size or number.

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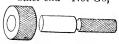
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Feeler or Thickness Gage — Has blades of different thicknesses, in thousandths of an inch, so that slight variations can be felt or measured.

Gear Tooth Depth Gage — A gage for measuring the depth of gear teeth. Requires a different gage for each pitch.

Limit Gage — A plug or other gage having one end larger and the other smaller than the nominal size. If the small end of the plug goes in but the large

end does not, the size is between the two and within the limits of the gage. Similarly, in the case of a female limit gage, if the large end of the gage goes over the piece of work and the small end does not go over it, the work is within the established limits. Ordinarily, one end of a limit gage is marked "Go," and the other end "Not Go," or else they are stamped + and -.



Plug and Ring Gage — Gages for use in measuring inside and outside work or for use in setting calipers.

Radius or Curve Gage — Made like a feeler or thread gage but has each blade with a given outside radius on one end and inside a radius on the other for gaging small filiets or round edges.

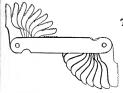
Scratch Gage — For scratching a line at
a given distance from one side of
a piece. Adjustable for different
lengths.

Snap Gage — A solid caliper used for either inside or outside measurement. This shows a combined gage for outside and inside work. Sizes can be the same or give the allowance for any kind of fit desired.

Splining or Key-seat Gage — Gage for laying out key-seats on shafts.



Surface Gage — A tool for gaging the hight between a flat surface such as a planer table or a surface plate and some point on the work. This can then be transfered to any other point.

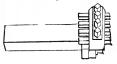


Thread Gage — Tool with a number of blades, each having the same number of notches per inch as the thread it represents. Made for different kinds of threads and in various forms.



Wire Gage — Gage for measuring sizes of wire. The wire fits between the sides of the opening, not in the holes. Sometimes made in the form of a circular disk.

Worm Thread Tool Gage — For grinding thread tool for worm threads — 20 degree angle.



Gang Tool — A holder with a number of tools, generally used in the planer but sometimes in the lathe. Each tool cuts a little deeper than the one ahead of it.

GEARS

Angular Gears — Sometimes applied to bevel gears and also to spur gears with helical or skew teeth. See those terms for definition.

Annular Gear — Toothed ring for use in universal chucks and similar places. Teeth can be on any of the four faces although when inside it is usually called an internal gear.



Bevel Gears — Gears cut on conical surfaces to transmit power with shafts at an angle to each other. When made for shafts at right angles and with both gears of the same size are often called "miter" gears. Teeth may be either straight, skew or herring bone.

Crown Gear — A gear with teeth on the side of rim. Used before facilities for cutting bevel gears existed. Seldom found now.



Elliptical or Eccentric Gears — Gears in which the shaft is not in the center.

May be of almost any shape, oval, heart-shape, etc. Printing presses usually have good examples of this.

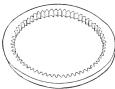
Helical Gears — Gears having teeth at an angle across the face to give a more constant pull. Also give side thrust. More often called "skew" teeth.



Herring-bone Gears — Gears having teeth cut at a double angle. Made by putting two helical or "skew" tooth gears together. Does away with side or end thrust.



Intermittent Gears — Gears where the teeth are not continuous but have plain surfaces between. On the driven gear these plain surfaces are concave to fit the plain surface of the driver and the driven wheel is stationary while the plain surfaces are in contact.



Internal Gears — Gears having teeth on the inside of a ring or shell.

Module or Metric Gears — French system of making gears with metric measurement. Pitch diameter in millimeters divided by the number of teeth in the gear.

Pin Gear — Gear with teeth formed by pins such as the old lantern pinion. Also formed by short projecting pins or knobs and only used now in some feeding devices.

Quill Gears - Gears or pinions cut on a quill or sleeve.

Skew Gears — See Helical.



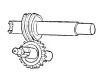
Spiral Gears — Spur gears with spiral teeth which run together at an angle and do the work of bevel gears.



Spur Gears — Wheels or cylinders whose shafts are parallel, having teeth across face. Teeth can be straight, helical or skew or herring bone.

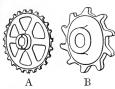
Staggered Tooth Gears — Made up of two or more straight tooth spur gears, teeth set so that teeth and spaces break joints instead of presenting a continuous pull.

Worm Gears — Spur gears with teeth cut on angle to be driven by a worm. Teeth are usually cut out with a hob to fit the worm.









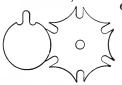
Sprocket Gears — Toothed wheels for chain driving. A is the regular and B is a hook tooth for running one way only.

Gear Teeth — The projections which, meshing together, transmit a positive motion. The involute curve tooth is now almost universal. The older form has a 11½ degree pressure angle but some are using a shorter tooth, known as a "stub" tooth, with 20 degrees pressure angle. An involute tooth rack has straight sides to the teeth.

Gears, Pitch of — Chordal, distance from center of one tooth to center of next in a direct line.

Circular, distance from center of one tooth to center of next along the pitch line.

Diametral, number of teeth per inch of diameter.



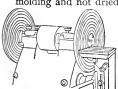
Geneva Motion — A device which gives a positive but intermittent motion to the driven wheel but prevents its moving in either direction without the driver. The driver may have one tooth as shown or a number if desired. Also made so as to prevent a complete revolution of the driven wheel.

German Silver — An alloy of copper 60 parts, zinc 20 parts, nickel 20 parts.

Gib — A piece located alongside a sliding member to take up wear.

Gland — A cylindrical piece enveloping a stem and used in a stuffing-box to make a tight joint.

Green Sand Molds — Molds made of sand that is moistened for molding and not dried out or baked before pouring.

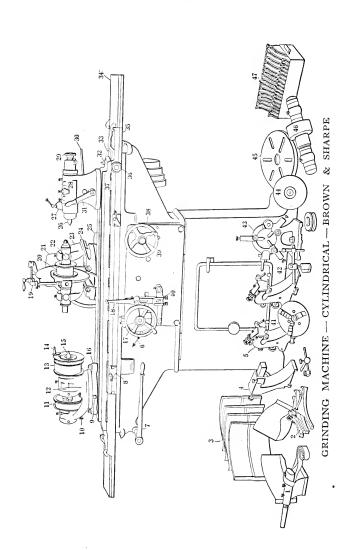


Grinder, Disk — A grinding machine having steel disks which are covered with emery cloth. Some disks have spiral grooves to give cushions under the emery cloth.

Grinder Wheel Dresser -



 A tool consisting of pointed or corrugated disks of hard metal which really break or pry off small particles of the grinding wheel when held against its rapidly reolving surface.

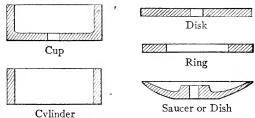


Grinding Machine --- Parts of

- 1. Internal grinding fixture.
- 2. Water guard supports.
- 3. Water guards.
- 4. Plain back rests.
- Universal back rest.
- 6. Automatic cross-feed pawl.
- 7. Starting and stopping lever.
- 8. Table-reversing dogs.
- Headstock index finger.
 Live spindle-locking pin.
- 11. Live spindle-driving pulley.
- 12. Headstock.
- 13. Dead center pulley.
- 14. Work driving arm and pin.
- Headstock center.
 Headstock base.
- 17. Cross-feed hand wheel.
- 18. Reversing lever.
- 19. Water piping.
- 20. Wheel-driving pulley.
- 21. Wheel guards.
- 22. Spindle box.
- 23. Wheel stand.
- 24. Wheel-stand platen.

- 25. Wheel-stand slide.
- 26. Footstock center.
- 27. Diamond tool-holder.
- 28. Footstock.
- 29. Tension adjusting knob.
- 30. Quick-adjusting lever.
- 31. Clamping lever 32. Clamping bolt.
- 33. Table scale.
- 34. Bed water guard.
- 35. Sliding table.
- 36. Swivel table knob.
- 37. Swivel table.
- 38. Hand wheel.
- 39. Table travel control.
- 40. Automatic cross-feed.
- 41. Universal chuck.
- 42. Tooth rest.
- 43. Center rest.
- 44. Face-grinding chuck.
- 45. Face plate.
- 46. Internal grinding counter.
- Work-driving dogs.

Grinding Wheels — Common types of grinding wheels made of emery, corundum, carborundum and alundum, are the disk, ring, saucer, cup and cylinder. Disk and ring wheels are used on the periphery; saucer wheels on the thin edge; cup and cylinder wheels on the end. The latter are commonly used for surface grinding. See pages 224 and 225 for other shapes.



Gripe — Local name for machine clamp.

Ground joint — A joint finished by grinding the two parts together with emery and oil or by other abrasives.

Ground — A connection between the electric circuit and the earth.

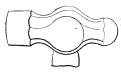
Gudgeon — Local name for a trunnion or bearing which projects from a piece as a cannon.



Guide Liner - A tool for use in locomotive work for lining up guides and cross heads.

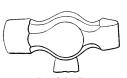
H

Half Nut — A nut which is split lengthwise. Sometimes half is used and rides on screw, in others both halves clamp around screw as in the half nut of a lathe carriage.

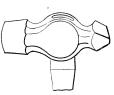


Hammer — The common types of machinists' hammers are the ball peen, straight peen and cross peen, as shown. The so-called engineer's and the riveting hammers have cross peens.

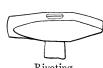
Ball Peen



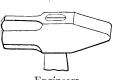
Straight Peen



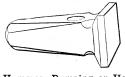
Cross Peen



Riveting



Engineers



Hammer, Blacksmith's Flatter - A flatfaced hammer used to smooth the surfaces of forgings. Is held on the work and struck by a helper with a sledge.

Hammer, Bumping or Horning - For closing seams on large cans, buckets, etc.

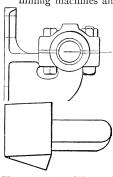
Hammer, Drop - Hammer head or "monkey" or "drop" is raised by hand or power and falls by gravity. Sometimes raised by a board attached to top of hammer head and running between pullevs. Others use a belt.

Hammer, Helve — Power hammer in which there is an arm pivoted in the center and power applied at the back end while the hammer is at the other and strikes the work on an anvil.

Hammer, Lever Trip — Trips the hammer by a cam or lever and allows it to fall.

Hammer, Spring — Comparatively small hammer giving a great variety in the force of blow. This is controlled by pressure of foot on lever.

Hand Wheels, Clutched — Hand wheels connected to shaft by a clutch which can be thrown out by a knob or otherwise so that accidental movement of wheel will not disturb setting. Used on milling machines and similar places.



Hanger, Drop — Shaft hanger to be fastened to ceiling with bearing held in lower end.

Hanger, Post — Shafting hanger for fastening to posts or other vertical structures.

Hardie — Blacksmith's cutting chisel which fits a hole in the anvil and forms the lower tool in cutting off work.

Harveyizing — The surface hardening of steel armor plates by using a bed of charcoal over the work and then gas turned on so it will soak in from the top. Not adapted to small work.

Hindley Worm — See Worm.

Hoist, Chain — Hoist with chain passing through pulley block used for hoisting.

Holder, Drill — Device for holding drill stationary while work is revolved by lathe chuck, or face place. Not a drill chuck.

Hooks, Twin or Sister — Double crane hook which resembles an anchor and allows load to be carried on either side.

Hub — A master punch used in making jewelry dies for fancy embossing, and various forms to which articles of gold and silver are to be struck.

Hunting Tooth — An extra tooth in a wheel to give it one more tooth than its mate in order to prevent the same teeth from meshing together all the time.

T

Idler or Idler Pulley — See Pulley, Idler.

Incandescent — A substance heated to white heat as in the bulb of a lamp.

Indexing, Compound — Indexing by combination of two settings of

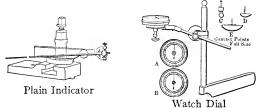
index, either by adding or subtracting.

Indexing, Differential — Indexing with the index plate geared to the spindle, thus giving a differential motion that allows the indexing to be done with one circle of holes and with the index crank turned in the same direction, as in plain indexing.

Indexing, Direct — Indexing work by direct use of dividing head of

milling machine.

Indicator, Lathe Test — Instrument with multiplying levers which shows slight variations in the truth of revolving work. Used for setting work in lathe or on face-plate.



Induction Motor — A motor which runs by the magnetic pull through the air without contact. Usually a constant-speed motor.

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Jack, Hydraulic — Device for raising weight or exerting pressure by pumping oil or other liquid under a piston or ram.

> Jack, Leveling — Small jacks (usually screw jacks) for leveling and holding work on planer beds and similar places. Practically adjustable blocking.

Jack, Screw — Device for elevating weights by

means of a screw.

Jack Shaft — See Shaft, Jack.

Jam Plates — Old name for screw plates and in many cases a true one as the thread was jammed instead of cut.

Jig, Drill — A device for holding work while drilling, having bushings through which the drill is guided so that the holes are correctly located in the piece. Milling and planing jigs (commonly called fixtures) hold work while it is machined in the milling machine and planer. Parts produced in jigs and fixtures are interchangeable.



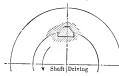
Joint, Universal — Shaft connection which allows freedom in any direction and still conveys a positive motion. Most of them can transmit power through any angle up to 45 degrees.

Journal Box — The part of a bearing in which the shaft revolves.

K

Kerf — The slot or passageway cut by a saw.

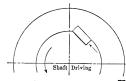
Key — The piece used to fasten any hollow object to a shaft or rod. Usually applied to fastening pulleys and fly-wheels to shafts; or locomotive driving wheels to their axles. Keys may be square, rectangular, round or other shape and fasten in any way. Are usually rectangular and run lengthwise of shaft.



Key, Barth — This key was invented several years ago by Carl G. Barth. It is simply a rectangular key with one-half of both sides beveled off at 45 degrees. The key need not fit tightly, as the pressure tends to drive it better into its seat. As a feather

key this key has been used in a great many cases to replace rectangular feather keys which have given trouble. It has also been used to replace keys which were sheared off under heavy load.

Key, Center — A flat piece of steel, with tapered sides, for removing taper shank drills from drill spindle or similar work.



Key, Lewis — A key invented by Wilfred Lewis about 20 years ago. Its position is such that it is subjected to compression only.



Key, Round End — Is fitted into a shaft by end milling a seat into which the key is secured. Where a key of some length is fixed in the shaft and a member arranged to slide thereon it is called a feather or feather key.

Key Seater — Machine for cutting keyways in shafts or hubs of pulleys or gears.



Key, Taper — The taper key is made with and without head. The taper is commonly ¹/₈ or ³/₁₆ inch per foot.

Key, Woodruff — A semi-circular key used in various kinds of shafts, studs, etc. It is fitted in place by merely sinking a seat with a shank mill such as the Whitney cutter.

Keyway - A groove, usually square or rectangular, in which the key is driven or in which a "feather" slides. The groove in both the shaft and piece which is to be fastened to it, or guided on it, is called a keyway.

Knurling - See Nurling.

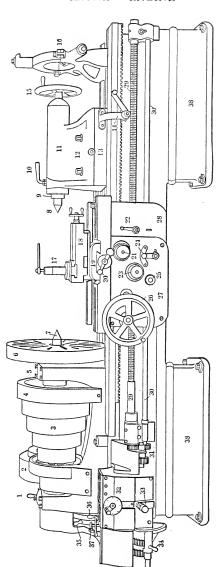
Τ.

- Land Space between flutes or grooves in drills, taps, reamers or other tools.
- Lap Applied to seams which lap each other. To the distance a valve must move before opening its port when valve is central on seat. To a tool usually consisting of lead, iron or copper charged with abrasive for fine grinding. See Lap, Lead.
- Lap Cutter For preparing the ends of band-saws with bands for brazing. Uses milling cutters.
- Lap Grinder This prepares the laps of band-saws by grinding.
- Lap, Lead Usually a bar of lead or covered with lead, a trifle smaller than the hole to be ground. Emery or some fine abrasive is used which gives a fine surface. Laps are sometimes held in the hand or are run in a machine and the work held stationary. Also consists at times of a lead-covered disk, revolving horizontally, which is used for grinding flat surfaces. Very similar in action to a potter's wheel.
- Lathe, Double Spindle Has two working spindles, so located that one gives a much larger swing than the other, and both can be used to advantage. Especially good for repair shops.
- Lathe, Engine The ordinary form of lathe with lead screw, power feed, etc.

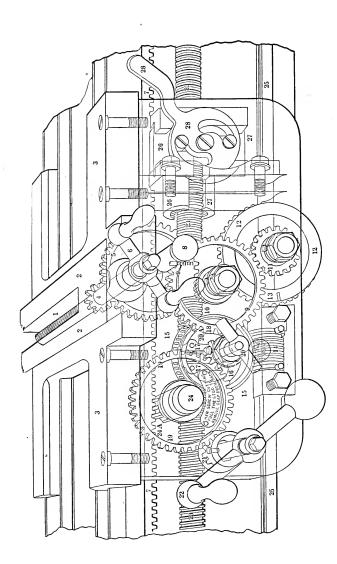
Lathe, Engine - Parts of

- Rear bearing.
- Back-gear case.
- Cone pulley.
- 4. Face-gear guard.
- 5. Front bearing.
- 6. Face plate.
- 7. Live center.
- 8. Dead center.
- 9. Tail spindle.
- 10. Tail-spindle lock.
- 11. Tailstock slide.
- 12. Locking bolts.
- Tailstock base.
- 14. Tailstock pinion.
- 15. Tailstock hand wheel.
- 16. Steady rest.
- Tool post.
- Compound rest.
 - 19. Cross-slide.

- Cross-feed screw.
- 21. Cross-power feed.
- 22. Half-nut handle.
- 23. Regular power feed.
- 24. Feed reverse.
- 25. Gear stud.
- 26. Hand feed.
- 27. Front apron.
- 28. Rear apron.
- 29. Lead screw.
- 30. Feed rod.
- 31. Feed gears.
- 32. Feed box.
- 33. Change gear handle.
- 34. Compound gears.
- 35. Change-gear handle.
- 36. Change-gear handle.
- 37. Change-gear handle.
 - 38. Bed.



LATHE—ENGINE—SCHUMACHER & BOYE



Lathe Apron, Reed - Parts of

1. Cross-feed screw.

2. Cross-slides.

3. Wing of saddle.

4. Cross-feed pinion.

5. Cross-feed gear.

6. Cross-feed handle.

Rack.

8. Power cross-feed and control.

9. Gear in train.

10. Pinion for cross-feed.

11. Main driving pinion.

12. Bevel gear.

13. Bevel pinion.

14. Feed-worm.

15. Feed-worm wheel.

16. Clutch ring.

17. Clutch levers.

18. Pinion.

19. Gear in train.

20. Feed-clutch handle.

20A. Clutch spreader.

21. Hand pinion. 22. Carriage handle.

23. Lead screw.

24. Rack pinion knob.

24A Rack pinion.

25. Feed rod.

26. Upper-half nut. 27. Lower-half nut.

28. Half-nut cam.

Note:—Cross-feed is from bevel pinion 13, through gears 12, 11, 9, 10, and 4. Regular feed is through worm 14, worm wheel 15, clutch 16, pinion 18, gears 19 and 24A. Hand movement is through handle 22, pinion 21, 19 and 24A.

Lathe, Extension — So made that bed can be lengthened or shortened. When bed is made longer, there is a gap near head, increasing the

swing for face-plate work.

Lathe, Fox — Brass workers' lathe having a "fox" or chasing bar for cutting threads. The bar has a "leader" which acts as a nut on a short lead screw or "hob" of the desired pitch (or half the pitch if the hob is geared down 2 to 1) and carries a tool along at the right feed for the thread. Sometimes has a turret on the back head.

Lathe, Gap — Has V-shaped gap in front of head stock to increase

swing for face-plate work.

Lathe, Gun — For boring and turning cannons and rapid-fire guns. Lathe, Ingot — For boring, turning and cutting off steel ingots.

Lathe, Precision — Bench lathe made especially for small and very accurate die, jig or model work.

accurate die, jig or model work.

Lathe, Projectile.—Simply a heavy lathe for turning up projectiles. Sometimes has attachment for pointing them.

Lathe, Pulley - Especially designed for turning pulleys, can turn

them crowning or straight.

Lathe, Roll Turning — For turning rolling mill, steel mill and calendar rolls.

Lathe Screw Cutting — Having lead screw and change grows for

Lathe, Screw Cutting — Having lead screw and change gears for cutting threads.

Lathe, Shafting — For turning long shafts or similar work.

Lathe, Speed — A simple lathe with no mechanically actuated carriage or attachments.

Lathe, Spinning — For forming sheet metal into various hollow shapes, all circular. Done by forcing against a form of some kind (with a single round ended tool) while it is revolving.

Lathe, Stone Turning — Specially designed for turning stone columns or similar shapes.

Lathe, Turret — Having a multiple tool holder which revolves.

This is the turret. Usually takes place of tail or foot stock but not always. Usually has automatic devices for turning turrer and sometimes for feeding tools against work.

Lathe, Vertical — Name given one type of Bullard boring mill on account of a side head which acts very much like a lathe carriage and does a large variety of work that would ordinarily be done on the face-plate of a lathe.

Lathe, Watchmaker's - A very small precision lathe.

Lead — The advance made by one turn of a screw. Often confused with pitch of thread but not the same unless in the case of a single thread. With a double thread the lead is twice as much as the pitch.

Level — Instrument with a glass tube or vial containing a liquid which does not quite fill it. The tube is usually ground on an arc so that bubble can easily get to the center. Alcohol is generally used as it does not freeze at ordinary temperatures.

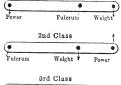
Level, Engineers' — Level mounted on tripod and having telescope for leveling distant objects.

Level, Pocket - Small level to be carried in pocket.

Level, Quartering — A tool for testing driving wheels to see if crank-

Ph H H ee tt si tt si tt

pins are set 90 degrees apart. The level has a forked end and with the angles shown. Placing this on the crank-pin and lining the edge with the center of axle should bring the bubble of level in the center. If the same result is obtained on the other wheel the crank-pins are 90 degrees apart.



Levers — Arms pivoted or bearing on points called fulcrums. Divided into three classes as shown. First has fulcrum or bearing point between power and weight, second has weight between power and fulcrum and third has power between weight and fulcrum.

Line Shaft - See Shaft, Line.

Liner — A piece for separating pieces a desired distance; also called shim.

Live Center - See Center, Live.

Loam Mold — Made with a mixture of coarse sand and loam into a sort of plaster which is spread over brick or other framework to make the mold. Used on large castings to produce a smoother finish than is to be had with green sand.

M

Machinists' Clamp - See Clamp.

Magnet Electro — Usually a bar of iron having coils of insulated wire around it which carry current. Permanent magnets are of hardened steel with no wire or current around them.

Mandrel — See Arbor.

Marking Machine — For stamping trade-marks, patent dates, etc., on cutlery, gun barrels, etc. Stamps are usually on rolls and rolled into work.

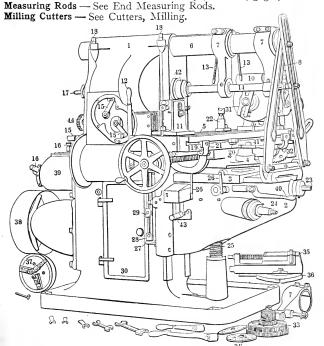
Master Die — A die made standard and used only for reference

purposes or for threading taps. **Master Plate** — See Plate, Master.

Master Tap — A tap cut to standard dimensions and used only for reference purposes or for tapping master dies.

Match Board — The board used to hold patterns, half on each side, while being molded on some types of molding machines.

Measuring Machine — Practically a large bench caliper of any desired form to measure work such as taps, reamers, gages, etc.



MILLING MACHINE - UNIVERSAL - MILWAUKEE

Milling Machine - Universal - Parts of

- Column.
- 2. Knee.
- Saddle.
- Swivel carriage.
- 5. Work table.
- 6. Over arm.
- Arm brackets (arbor supports.) 8. Arm braces (harness).
- 9. Knee clamp (for arm braces).
- 10. Spiral dividing head.
- Tailstock.
- Starting lever.
- 13. Oil tubes.
- Cutter arbor.
- 15. Speed-changing lever.
- Feed-changing lever.
- 17. Draw-in rod for arbor.
- 18. Arm-clamp screws.
- 19. Table stops.
- 20. Table-feed trip block.
- Fixed table-feed trips.
- 22. Steady rest.
- 23. Cross-feed screw.

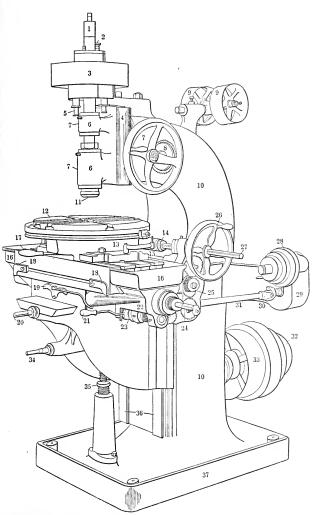
- Elevating shaft.
- 25. Elevating screw (telescopic).
- Saddle-clamp levers. Knee-clamp levers.
- 28. Fixed vertical feed trip.
- Vertical feed-trip blocks.
- 30. Door. 31. Dog driver.
- 32. Change-gear bracket.
- 33. Change gears.
- 34. Index plates.
- 35. Vise.
- 36. Swivel base. Universal chuck.
- 38. Driving pulley.
- Feed box.
- 40. Cross and vertical feed handle,
 - Table-feed handle. Clutch-drive collar.
 - 43. Interlocking lever to prevent the engagement of more than one feed at a time.

may be done. Equipped with index centers, chuck, etc. Milling Machine, Vertical — Has a vertical spindle for carrying cutter. Milling Machine -- Vertical -- Parts of

Milling Machine — Operating tool is a revolving cutter. Has table for carrying work and moving it so as to feed against cutter. Milling Machine, Universal — Has work table and feeds so arranged that all classes of plane, circular, helical, index, or other milling

- Spindle drawbar cap.
- Back-gear pull pin.
- Spindle-driving pulley. Spindle head.
- 5. Back-gear sliding pinion and stem gear.
- 6. Spindle upper box.
- Spindle lower box.
- 7. Spindle head bearing.
- 8. Head-feed gear.
- Idler pulleys.
- 10. Standard.
- 11. Spindle.
- 12. Rotary attachment.
- 13. Rotary attachment feed-trip dog and lever.
- 14. Rotary attachment feed clutch.
- Rotary attachment base.
- Table and table oil pans.
- Rotary attachment binder.

- 18. Feed-trip dogs, right and left.
- 19. Feed-trip plate.
- Cross-feed screw. 21. Feed-clutch lever.
- 22. Carriage.
- 23. Feed clutch.
- Table-feed screw.
- 25. Rotary attachment feed gears. 26. Rotary attachment hand wheel.
- Rotary attachment feed rod.
- 28. Feed-driving cone.
- 20. Feed bracket. 30. Universal joint.
- Telescopic feed shaft.
- 32. Driving cone.
- 33. Driving pulley. 34. Knee-elevating shaft.
- Knee-elevating telescopic screw.
- 36. Face of standard.
- Base.



MILLING MACHINE - VERTICAL - BECKER

Miter — A bevel of 45 degrees.

Mold — The mold consists of the cope and the drag or nowel, with the sand inside molded to pattern and ready to pour.

Mold Board - The board used to put over a flask to keep sand from falling when being handled and sometimes used to clamp

on when fastening molds together. Mufflers - Ovens or furnaces, usually of clay, where direct heat is

not required. Muley Belt - See Belt, Muley. Muley Shaft - See Shaft, Muley.

N

Necking Tool - Tool for turning a groove or neck in a piece of

Nose — In shop work applied to the business end of tools or things. The threaded end of a lathe or milling-machine spindle or the end of "hog nose drill" or similar tocl.

Nowel — Same as Drag.

Nurling — The rolling of depressions of various kinds into metal by the use of revolving hardened steel wheels pressed against the work. The design on the nurl will be reproduced on the work. Generally used to give a roughened surface for turning a nut or screw by hand.

Nut, Cold Punched - A nut punched from flat bar stock. The hole

is usually reamed to size before tapping.

Nut, Hot Pressed - A nut formed hot in a forging machine.

Nut, Castellated or Castle — A nut with slot across the face to admit a cotter pin for locking in place.

Nut Machine - For cutting, drilling and tapping nuts from a bar or rod.

Nut Tapper - For tapping hole in nuts.



Nuts - See Bolts.

Nut, Wing - A nut operated by hand and very commonly used where a light and quick clamping action is required.

0

Ogee - Name given to a finish or beading consisting of a reverse curve. Applied to work of any class, wood or iron.

Ohm - The unit of electrical resistance. One volt will force one ampere through a resistance of one ohm.

Oval - Continuously curved but not round, as a circle which has been more or less flattened.

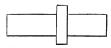
Pawl — A hinged piece which engages teeth in a gear, rack or ratchet for moving it or for arresting its motion. Sometimes used to

designate a piece such as a reversing dog on a planer or milling machine.

Peening — The stretching of metal by hammering or rolling the surface. Used to stretch babbitt to fit tightly in a bearing, to straighten bars by stretching the short or concave side, etc.

Pickling Castings — Dipping into acid solution to soften scale and remove sand. Solution of three or four parts of water to one of sulphuric acid is used for iron. For brass use five parts water to one of nitric acid.

Pin, Collar — A collar pin is driven tight into a machine frame or member and adapted to carry a roll, gear, or other part at the



outer end. It differs from the collar stud in not having a thread at the inner end. When drilled through the end for a cotter pin it is known as a fulcrum pin, as it is then especially suited for carrying rocker arms, etc.

Pin, Dowel, Screw — Dowel pins are customarily made straight, or plain taper and fitted into reamed holes. When applied in such a position in a mechanism that it is impossible to remove them by driving out, they are sometimes threaded and screwed into place.

Pin, Taper — Taper pins for dowels and other purposes are regularly manufactured with a taper of ¼ inch to the foot and from ¾ to



6 inches long, the diameters at the large end of the sizes in the series ranging from about $\frac{5}{32}$ to $\frac{23}{32}$ inch. The reamers for these pins are so proportioned that each

size "overlaps" the next smaller size by about ½ inch.

Pickling Forgings — Putting in bath of 1 part sulphuric acid to 25 parts boiling water to remove scale. Can be done in 10 minutes. Rinse in boiling water and they will dry before rusting.

Pitch — The distance from the center of one screw thread, or gear tooth or serration of any kind to the center of the next. In screws with a single thread the pitch is the same as the lead but not otherwise.

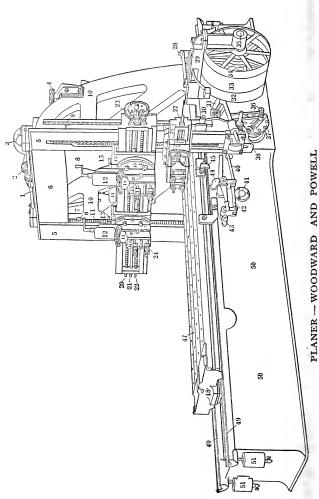


Pillow Blocks — Low shaft bearings, resting on foundations, or floors or other supports.

Pitman — A connecting rod; term used more commonly in connection with agricultural implements.

Planchet — Blank piece of metal punched out of a sheet before being finished by further work. Such as the blanks from which coins are made.

Planer — For producing plane surfaces on metals. Work is held on table or platen which runs back and forth under the tool which is stationary.



Planer - Parts of

 26. Side head. 27. Side head for feed screw. 28. Belt shifter. 29. Drive-pulley support. 30. Connection to feed rack. 	31. Regulator for vertical feed. 32. Forward driving pulley. 33. Loves pulley. 34. Return Iriving pulley. 35. Driving shaft.	36. Regulator cross-feed. 37. Cross-feed drive. 38. Vertical feed pinion. 39. Vertical feed rod. 40. Rod to belt shifter for reversing. 41. Bull or driving-wheel shaft.	 42. Connections to safety lock. 43. Lock to prevent table being moved. 44. Reversing latch or trip. 45. Forward stop dog. 46. Backward stop dog. 	 47. Platen or table. 48. Rack under platen. 49. Ways or V's. 50. Bed. 51. Oil reservoirs.
 Shaft for raising cross-rail. Gears for raising cross-rail. Pulley drive for raising cross-rail. Chain for counterweighting the cross-rail. 	5. Face of uprights. 6. Tie piece between uprights. 7. Handle controlling. 8. Crank handle for raising tool block. 9. Dood for moving food screw in cross-rail.	10. Upright or housing. 11. Screw for elevating cross-rail. 12. Tool slide. 13. Screw to clamp saddle to cross-rail. 14. Counterweight for left side of cross-rail. 15. Saddle.	 16. Swrvel. 17. Clamping bolt. 18. Clapper bock. 19. Clapper block. 20. Feed screw for left-hand head. 	 21. Vertical feed rod. 22. Feed screw for right-hand head. 23. Feed mechanism on end of cross-rail. 24. Tool-holding straps. 25. Clapper block pin.



Planer Centers — A pair of index centers to hold work for planing. Similar to plain milling machine centers.

Planer, Daniels — Wood planer with a table for carrying work under a two-armed knife swung horizontally from a vertical spindle. Very similar to a vertical spindle-milling machine of the planer type. Excellent for taking warp or wind out of lumber.

Planer, Open Side — A planer with only one upright or housing, supporting an overhanging arm which takes the place of the usual cross rail. Useful in planing work too wide to go in the ordinary planer.

Planer, Radius — For planing parts of circles such as links for locomotive or stationary engine valve motion.

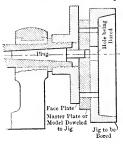
Planer, Rotary — Really a large milling machine in which the work is carried past a rotary cutter by the platen.

Planer Tools - See Tools, Planer.

Planer, Traveling Head — Planer in which work is stationary and tool moves over it.

Planishing — The finishing of sheet metal by hammering with smooth faced hammers or their equivalents.

Plate, Master - A steel plate serving as a model by which holes in



jigs, fixtures and other tools are accurately located for boring. In the illustration the piece to be bored is shown dowelled to the master plate which is mounted on the face plate of the lathe. In the master plate there are as many holes as are to be bored in the work, and the center distances are correct. The plate is located on a center plug fitting the lathe spindle, and after a given hole is bored in the work, the master plate and work are shifted and relocated with the center plug in the next hole in the plate and the corre-

sponding hole in the work is then bored out. This is one of the most accurate methods employed by the toolmaker on precision work.

Platen - A work holding table on miller, planer or drill.

Plumb Bob, Mercury — Plumb bob filled with mercury to secure weight in a small space.

Potter's Wheel — Probably the oldest machine known. Consists of a vertical shaft with a disk mounted horizontally at the top. The potter puts a lump of clay in the center, revolves the wheel with his foot or by power, and shapes the revolving clay as desired. See Lap, Lead for modern application of this in machine work.

Press, Blanking - For heavy punching or swaging.

Press, Broaching — A press for forcing broaches through holes in metal work. Usually cleans out or forms holes that are not round.

Press, Cabbaging — For compressing loose sheet metal scrap into convenient form for handling and remelting.

Press, Coining — For making metal planchets from which coins are stamped.

Press, Double Action — Has a telescoping ram or one ram inside the other, each driven by an independent cam so that one motion follows the other and performs two operations for each revolution of the press.

Press Fit - See Fit.

Press, Forcing — For forcing one piece into another, such as a rod brass into a rod.

Press, Forging — For forging metal by subjecting it to heavy pressure between formers or dies.

Press, Horning — For closing side seams on pieced tinware.

Press, Inclinable — One that can be used in vertical or inclined position.

Press, Screw — Pressure is applied by screw. Heavier work is done in this way than by foot or hand press.

Press, Straight-sided - Made with perfectly straight sides so as to give great strength and rigidity for heavy work.

Press, Pendulum — Foot press having a pendulum like lever for

applying power to the ram.

Profiling Machine - Has rotary cutter that can be made to follow outline or pattern in shaping small parts of machines. Practically a vertical milling machine.

Protractor, Bevel - Graduated semicircular protractor having a pivoted arm for measuring off angles.

Pull-pin — A means of locking or unlocking two parts of machinery. Sometimes slides gears in or out of mesh and at others operates a sliding key which engages any desired gear of a number on stud.

Pulley, Gallow or Guide — Loose pulley mounted in movable frames

to guide and tighten belts.

Pulley, Idler - A pulley running loose on a shaft and driving no machinery, merely guiding the belt. Practically same as a "loose pulley."

Pulley, Loose — Pulley running loosely on shaft doing no work. Carries belt when not driving tight (or fast, or working) pulley. Used on countershafts, planers, grinders, etc., where machine is idle part of time. Belt is then on the loose pulley but when shifted to tight pulley the machine starts up. See Belt Shifter, Friction Clutch.

Punching - A piece cut out of sheet stock by punch and die; the same as blank.

Punch, Belt — Hollow, round or elliptical punch for cutting holes for belt lacing.

O

- Quadrant A piece forming a quarter circle. A segment of a circle.

 The swinging plate carrying the change gears in the feed train at the end of the lathe.
- Quick Return A mechanism employed in various machine tools to give a table, ram or other member a rapid movement during the return or non-cutting stroke.
- Quill A hollow shaft which revolves on a solid shaft, carrying pulleys or clutches. When clutches are closed the quill and shaft revolve together.

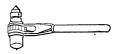
R



Rack — A strip cut with teeth so that a gear can mesh with it to convert rotary into reciprocating motion or vice versa.

Rack Cutter — Cuts regularly spaced teeth in a straight line. Cutting tool is either a milling cutter or a single point tool.

Ratchet — A gear with triangular shaped teeth adapted to be engaged by a pawl which either imparts intermittent motion to the ratchet or else locks it against backward movement when operated otherwise.



Ratchet Drill — Device for turning a drill when the handle cannot make a complete revolution. A pawl on the handle drops into a ratchet wheel on the barrel so that it can be turned one or more teeth.

Recess — A groove below the normal surface of work. On flat work a groove to allow tool to run into as a planer, or a slide to run over as a cross-head on a guide. In boring a groove inside a hole. If long it is often called a chamber.

Relief or Relieving — The removing of, or the amount removed to reduce friction back of cutting edge of a drill, reamer, tap, etc. Also applied to other than cutting tools. See "backing off."

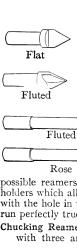
REAMERS

Reamer — A tool to enlarge a hole already existing, whether a cast or cored hole or one made by a drill or boring bar. Reamers are of many kinds and shape as indicated below. Usually a reamer gives the finishing touch to a hole.

Ball Reamer — Usually a fluted or rose reamer for making the female portion of a ball joint. It is considered advisable to space the teeth irregularly as this tends to prevent chattering.

Bridge Reamer — A reamer used by boiler-makers, bridge builders, ship-builders, etc., has a straight body from A to B and tapered

end from B to C. This reamer has a taper shank and can be used in an air drill.



Center Reamer — Center reamers, or countersinks for centering the ends of shafts, etc., are usually made 60 degrees included angle.

Chucking Reamer — Chucking reamers are used in turret machines. The plain, fluted type has teeth relieved the whole length; while the rose reamer cuts only on the end as there

Kose is no peripheral clearance. Where possible reamers used in the turret should be mounted in floating holders which allow the reamer to play sidewise sufficiently to line up with the hole in the work which may be so drilled or bored as not to run perfectly true prior to the reaming operation.

Chucking Reamer (Three-groove) — Spiral fluted chucking reamers with three and four grooves are employed for enlarging cored

holes, etc. They are also made with oil passages through them and in this form are adapted to operating in steel.

Flat Reamer — A reamer made of a flat piece of steel. Not much used except on brass work and then usually packed with wooden strips to fit the hole tightly. Flat reamers are not much used except for taper work.

Half-round Reamer — Used considerably in some classes of work, particularly in small sizes and taper work when taper is slight. Not much used in large sizes. Somewhat resembles the "hognose drill" in general appearance except that this is always quite short on cutting edge.

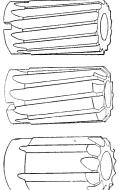
Straight

Threaded End

Spiral

Hand Reamer — Reamers enlarge and finish a hole produced by drilling, boring, etc. The cut should be light for hand reamers and the reamer held straight to avoid ruining the hole. The threaded end reamer has a fine thread to assist in drawing the reamer into the work. The spiral reamer is cut left-hand to prevent its drawing into the hole too rapidly. Reamers are slightly tapered at the point to enable them to enter.

Pipe Reamer (Briggs) — Pipe reamers to the Briggs standard taper of 3-inch per foot are used for reaming out work prior to tapping with the pipe tap.



Shell Reamer - Shell reamers have taper holes to fit the end of an arbor on which they are held in the chucking machine. They are made with both straight and spiral flutes.

Shell Reamer (Rose) - Rose reamers cut on the end only as there is no peripheral clearance. They are very accurate tools for finishing holes. shell reamers are made with taper holes to fit an arbor for holding them in the turret machine.

Taper Reamer - For finishing taper holes two or more reamers are sometimes used. The roughing reamer is often provided with



Finishing

nicked or stepped teeth to break up the Taper reamers are also made with spiral teeth. Where the taper is slight the spiral should be left-hand to prevent the reamer from drawing in too fast; where the taper is abrupt the teeth, if cut with right-hand spiral, will help hold the reamer to the cut and make the operation more satisfactory.

Taper Pin Reamer - Standard taper pin reamers are made 1-inch taper per foot and each size in the series will overlap the next size smaller

by about one-half inch.

Taper Reamer (Locomotive) - Reamers for locomotive taper pins have a taper of $\frac{1}{16}$ inch per foot.

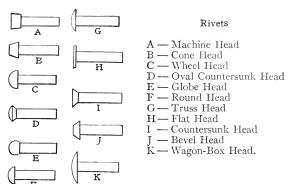
Rests, Slide — Detachable rests capable of being clamped to brass lathe bed at any desired point and usually arranged to give motion to tool in two ways; across the bed to reduce diameter or cut-off, and with the bed for turning. Invented by Henry Maudsley.

Rheostat — An adjustable resistance box so that part of the current can be cut out of the motor.

Riddle - Name given to a sieve used in foundries for siftings and for the molds.

Riffle - Name given a small file used by die sinkers and on similar work.

Rivet — A pin for holding two or more plates or pieces together. A head is formed on one end when made; the other end is upset after the rivet is put in place and draws the riveted members close together.



Rivet Machine - For making rivets from metal rods.

Roller Bearing - See Bearing.



Rule, Hook — Rule with a hook on the end for measuring through pulley holes and in similar places.

Rule, Key-seat — For laying out key-seats on shaft or in hubs.

Rule, Shrink — Graduated to allow for shrinkage in casting. Used by pattern-makers and varies with metal to be cast.

Run — Applied to drilling or reaming when the tool shows a tendency to leave the straight or direct path. Caused by one lip or cutting edge being less sharp than the other, being ground so one lip leads the other, or from uneven hardness of material being drilled.

Running or Sliding Fit - See Fits.

Rust Joint — A joint made by application of cast-iron turnings mixed with sal-ammoniac and sulphur to cause the turnings to rust and become a solid body.

S

Salamander — The mass of iron left in the hearth when a furnace is blown out for repairs.

Sand Blast — Sand is blown by compressed air through a hose as desired. Used to clean castings, stonework, etc.

Sanding Machine — A machine in which woodwork is finished by means of rolls or wheels covered with sandpaper.

Sanding Belt — Endless belt of some strong fabric, charged with glue and sand. For sandpapering wood held in hand or by clamps.

Saw, Band — Continuous metal band, toothed on one edge and guided between rolls. Mostly used on woodwork, but occasionally on metal work, especially in European shops.

Saw Bench, Universal — Bench on which lumber is brought to the saw for ripping, cross-cutting, dadoing, mitering, etc.

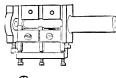
Saw, Cold — For sawing metal. Circular saws are generally used though not always, band saws being occasionally employed.



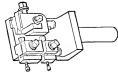
Saw, Hack — Close-toothed saw for cutting metal. Usually held in a hand frame but power hack saws are now becoming very common in shops.

Scarf — The bevel edge formed on a piece of metal which is to be lap-welded.

SCREW MACHINE TOOLS

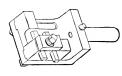


Box Tool, Bushing — The cutters in this tool are placed with edges radial to the stock and may be adjusted to turn the required diameter by the screws in the rear. The stock is supported in a bushing and must therefore be very true and accurate as to size.

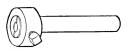


Box Tool, Finishing — The material turned in this box tool is supported by adjustable back rest jaws and the cutters are also adjustable in and out as well as lengthwise of the tool body.

Box Tool, Roughing — This tool has one or more cutters inverted over the work and with cutting edges tangent to the material.

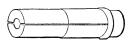


The back rest is bored out the size the screw or other piece is to be turned and the cutter turns the end of the piece to size before it enters the back rest. Sometimes a pointing tool is inserted in the shank for finishing the end of the work.

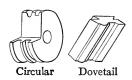


Drill Holder — The end of the drill holder is split and provided with a clamp collar by which the holder is closed on the drill.

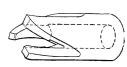
Feed Tube — The screw-machine feed tube or feed finger is closed prior to hardening and maintains at all times a grip on the stock.



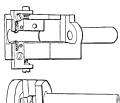
The rear end is threaded and screwed into the tube by which it is operated. It is drawn back over the stock and when the chuck opens is moved forward feeding the stock the right distance for the next piece.



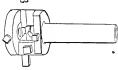
Forming Tools - Circular forming cutters are generally cut below center to give proper clearance and the tool post is bored a corresponding amount above center to bring the tool on the center line. Dovetail cutters are made at an angle of about 10 degrees for clearance.



Hollow Mill - Usually made with 3 prongs or cutting edges and with a slight taper inside toward the rear. A clamp collar is used on mill like a spring die collar and a reasonable amount of adjustment may be obtained by this collar. Hollow mills are frequently used in place of box tools for turning work in the screw machine.

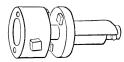


Nurling Tool - The two nurls in this box are adjustable to suit different diameters of work.



Pointing Tool - The bushing in this tool receives and supports the end of the round stock and the cutters carried in the frame form and point the end.

Revolving Die Holder - The common type of revolving die holder which is generally used with spring dies, has a pair of driving



pins behind the head and in the flange of the sleeve which fits into the turret hole. At the rear end of the sleeve is a cam surface which engages a pin in the shank of the head when the die The die is run on to the is reversed. work with the driving pins engaged. .

When the work is reversed, the cam at the rear engages the pin in the shank and holds the die from turning so that it must draw off the work.



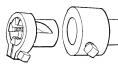
Spotting Tool — This tool spots a center in the end of the bar of stock to allow the drill to start properly, and also faces the end of the piece true.

times called "centering and facing" tool. It is desirable to have the included angle of the cutting point less than that of the drill which follows it in order that the latter may start true by cutting at the corners first.

Spring Collet - Spring collets or chucks are made to receive round,

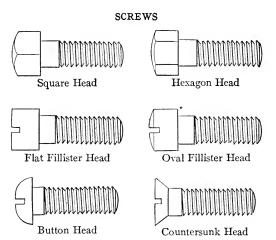


square, hexagonal or other stock worked in the screw machine. The collet is hardened and is closed in operation by being pressed into the conical cap into which it fits. When released it springs open sufficiently to free the stock and allow it to be fed through the collet.



Spring Die and Extension — Spring dies or prong dies are provided with a collar at the end for adjusting and are easily sharpened by grinding in the flutes.

Screw Plates — Holders for dies for cutting threads on bolts or screws. Dies are usually separate but sometimes cut in the piece which forms the holder.

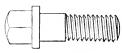


Cap Screws — Cap screws are machined straight from point to head, have finished heads and up to 4 inches in length are usually threaded three-fourths of the length. When longer than 4 inches they are threaded one half the length, which is measured under the head, except in the case of countersunk head screws which are measured over all. Cap-screw sizes vary by 16ths and 8ths and are regularly made up to 1 or 1½ inch diameter, while machine screws with which they are frequently confused are made to the machine-screw gage sizes.

Flat fillister heads on cap screws are often called "round" heads; oval fillister heads are frequently designated as "fillister"

heads, and countersunk heads as "flat" heads. When a countersunk or flat head has an oval top it is called a "French" head.

Fillister heads are also made with rounded corners as well as with the oval head shown above. Fillister head screws are known in England as cheese-head screws. The included angle of the countersunk or flat head is 70 degrees.



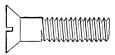
Collar Screw — Collar or collar head screws are used for much the same purposes as regular cap screws, and, in fact, are sometimes designated as "collar" cap or "collar head" cap screws.

Lag Screw — Lag screws, or coach screws, as they are often called, have a thread like a wood screw and a square or a hexagonal head. They are used for attaching countershaft hangers to over-head joists for fastening machines to wood

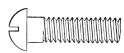
floors and for many other purposes where a heavy wood screw is required.



Machine Screws — Machine screws are made to the sizes of the machine-screw gage instead of running like cap screws in even fractions of an inch.



Counter Sunk or Flat Head



Button or Round

Set Screws — Set screws are threaded the full length of body and may or may not be necked under the head. They are usually case-hardened. Ordinarily the width of head across flats and the length of head are equal to the diameter of the screw. The headless set screw is known in England as a "grub" screw.

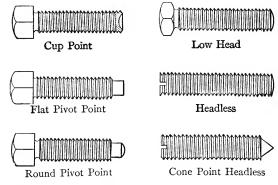


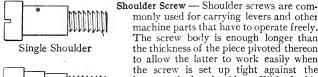


Round Point



Hanger Point

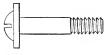




monly used for carrying levers and other machine parts that have to operate freely. The screw body is enough longer than the thickness of the piece pivoted thereon to allow the latter to work easily when the screw is set up tight against the With double bottom of the shoulder. shoulders two members may be mounted side by side and left free to operate independently of each other.



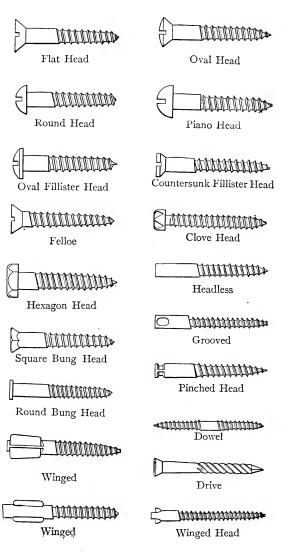
Thumb Screw - A screw with a winged or knurled head which may be operated by hand when a quick and light clamping effect is desired.



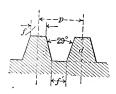
Double Shoulder

Washer-head Screw - The washer formed on this screw enables it to be used for holding pieces with large holes without applying a loose washer.

Wood Screws - Wood screws are made in an endless variety of forms, a number of which are shown on the following page. They range in size from No. o to No. 30 by the American Screw Company's gage and are regularly made in lengths from 1 inch to 6 inches. Generally the thread is cut about seven tenths of the total length of the screw. The flat-head wood screw has an included angle of head of 82 degrees.



Screw Thread, Acme 29 degree Standard -

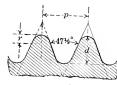


 $p = pitch = \frac{r}{no. threads per inch}$

 $d = depth = \frac{1}{2}p + .oro.$

f = flat on top of thread = p x .3707 The Acme screw thread is practically the same depth as the square thread and much stronger. It is used extensively for lead screws, feed screws, etc.

Screw Thread, British Association Standard—



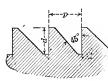
$$p = pitch$$

 $d = depth = p x .6$
 $r = radius = \frac{2 \times p}{11}$

This thread has been adopted in England for small screws used by opticians and in telegraph work, upon recom-

mendations made by the Committee of the British Association. The diameter and pitches in this system are in millimeters.

Screw Thread, Buttress -



$$p = pitch = \frac{1}{no. threads per inch}$$

 $d = depth = \frac{3}{4} p$

The buttress thread takes a bearing on one side only and is very strong in that direction. The ratchet thread is of practically the same form but sharper.

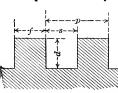
Screw Thread, International (Metric) Standard—

p = pitch $d = depth = p \times .6495$

$$f = flat = \frac{p}{8}$$

The International thread is of the same form as the Sellers or U. S. Standard. This system was recommended by a Con-

gress held at Zurich in 1898, and is much the same as the metric system of threads generally used in France. The sizes and pitches in the system are in millimeters.



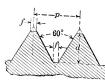
Screw Thread, Square ---

p=pitch= $\frac{1}{\text{no. threads per inch}}$ d=depth= $\frac{1}{2}$ p f=width of flat= $\frac{1}{2}$ p s=width of space= $\frac{1}{2}$ p.

While theoretically depth, width of space and thread are each one half

the pitch, in practice the groove is cut slightly wider and deeper.

Screw Thread, United States Standard -



$$p=pitch = \frac{1}{\text{no. threads per inch}}$$

$$d=depth = p \text{ x .6495}$$

$$f=flat = \frac{p}{8}$$

This thread was devised by Wm. Sellers,

and recommended by the Franklin Institute in 1869. It is called the U. S. Standard, the Franklin Institute, and the Sellers thread. The advantages of this thread are, that it is not easily injured, tap and dies will retain their size longer, and bolts and screws with this thread are stronger and better appearing. The system has been adopted by the United States Government, Master Mechanics and Master Car Builders' Associations, Machine Bolt Makers, and by many manufacturing establishments.

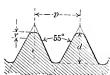
Screw Thread, V, 60 degree Sharp -



While the sharp V form gives a deeper thread than the U. S. Standard, the objections urged against the thread are, that the sharp top is injured by the

slightest accident, and, in the case of taps and dies, the fine edge is quickly lost, causing constant variation in fitting.

Screw Thread, Whitworth Standard -



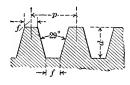
$$p=pitch = \frac{1}{\text{no. threads per inch}}$$

$$d=depth=p \times .64033$$

$$r=radius=p \times .1373$$

The Whitworth thread is the standard in use in England. It was devised by Sir Joseph Whitworth in 1841, the system then proposed by him

being slightly modified in 1857 and 1861.



$$p = pitch = \frac{r}{no. threads per inch}$$

 $d = depth = p \times .6866$

f = flat on top of thread = p x .335 This thread is commonly used in America for worms. It is considerably deeper than the Acme screw

thread of the same angle, namely 29 degrees.

Sector — A device used on an index plate of a dividing head for indicating the number of holes to be included at each advance of the index crank in dividing circles. The sector can be opened or closed to form as large or small an arc as necessary to cover the desired number of holes for each movement of the crank.

Set — The bend to one side of the teeth of a saw.

Set Screw - See Screws.

Shaft-bearing Stand — Shaft bearing which is fastened to floor.

Shaft Coupling - See Coupling.

Shaft, Flexible — Shaft made of a helical spring or of jointed parts, usually confined in a leather or fabric casing, to transmit power in varying directions.

Shaft, Jack — A secondary or auxiliary shaft, driven by the engine and in turn driving the dynamos or other machinery. Jack shafts are often introduced between a regular machine countershaft and the line shaft.

Shaft, Line — The shafting driving the machinery of a shop or section of a shop by means of pulleys and belts.

Shaft, Muley — A vertical shaft carrying two idler pulleys for carrying a belt around a corner. To be avoided where possible.

Shaper — Work is held on table or knee and tool moves across it, held by a tool post on the moving ram. Table adjustable for depth of cut, etc.

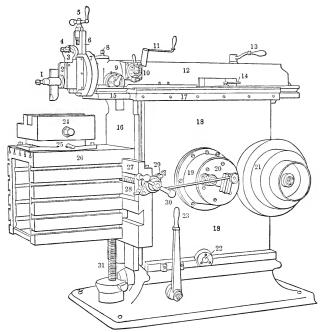
Shaper --- Parts of

- Tool post.
 Clapper block.
- 3. Clapper box.
- 4. Clamping bolts.
- 5. Down-feed screw.
- Tool slide.
- 7. Tool head.8. Binder for head.
- 9. Stop for down feed.
- 10. Down-feed adjustment.
- 11. Ram adjuster.
- 12. Ram.
- 13. Position lever.
- 14. Clamp for down feed.
- 15 Ram slide.
- 16. Face of column.

- 17. Ram guide.
- 18. Frame or body.
- 19. Feed box.
- 20. Feed regulator.
- 21. Cone-driving pulley.
- 22. Lever bearing
- 23. Power elevation of table.
- 24. Vise.
- 25. Swiveling base.
 - 26. Table.
- 27. Saddle.
 - 28. Cross-feed screw. 20. Cross-feed dog.
- 30. Cross-feed dog.
- 31. Elevating screw.

Shaper, Crank - Ram is driven by a crank motion.

Shaper, Draw Cut — Cutting stroke takes place when tool is moving toward frame of machine. This tends to draw the parts together.



SHAPER - POTTER AND JOHNSTON

- Shaper, Friction Ram is driven by rack and pinion through friction clutches. Ram is reversed by simultaneous release and engagement of these clutches. These are driven by open and crossed belts in opposite directions.
- Shaper, Gear Planes gear teeth by using a hardened cutter, shaped like a pinion gear, and moving across the face of the gear with a planing or shaping cut.
- Shaper, Geared Ram is driven by rack and pinion with a slow cutting stroke and a quick return by shifting open and crossed belts the same as on a planer.
- Shapers, Traverse or Traveling Head Ram feeds across work, which is stationary.
- Shear Tool for cutting metals between two blades. The name given to the way or V of a lathe or planer. A hoisting apparatus used on wharves or docks, consisting of two heavy struts like a long inverted V

Shears - The ways on which the lathe carriage and tail stock move are called "shears" by some, "ways" by others. They may be either V. flat or any other shape.

Shears, Alligator - Name given to machine where moving knife or

cutter works on a pivot.

Shears, Squaring — Has cutter bar guided at both ends.

Shears, Slitting - Arranged for slitting sheet metal. Rotary cutters are usually employed.

Shearing Machine - For cutting off rods, bars or plates.

Shedder — A plate or ring operated by springs or by a rubber pad to eject a blank from a compound die. It acts as an internal stripper, and is sometimes known as an ejector.

Sherardizing — The name given to a new process of dry galvanizing

of any iron product.

Shifter Forks - Arms to guide belt from tight to loose pulley or vice versa, by pressing the sides.

Shim — A liner or piece to place between surfaces to secure proper

adjustment. Shrink Fit — See Fits.

Slip Washer - See Open Washer.

Slotted Washer - See Open Washer.

Slotter — A machine for planing vertical surfaces or cutting slots. Tool travels vertically.

Socket, Grip - A device for driving drills and other tools with either a straight or taper shank.

Sow - In foundry work, the gate or central channel which feeds iron into the pigs when making pig iron.

Sow or Sow-block - Local name for a chuck for holding work, such

as dies. A ball chuck. Spinning — The forming of sheet metal by rolling it against forms such as lamp bodies. Lathes are made especially for this work.

Spline — Used in some sections in place of "key" and in others the

same as "feather." See Key and Feather.

Split Nut - Nut split lengthwise so as to open for quick adjustment. Spot or Spotting - Spotting is making a spot or flat surface for a set-screw point or to lay out from.



Spring, Compression - A helical spring which tends to shorten in action.

Spring, Helical — A spring coiled lengthwise of its axis like a screw thread. Often incorrectly called a spiral spring.

Spring, Leaf — A built up spring made of flat stock like a carriage spring or locomotive driving spring.



Spring, Spiral — A spring wound with one coil over the other as in a clock spring. Usually of flat stock, but not always.



Spring, Tension - A helical spring which tends to lengthen in action.



Spring, Torsion - A helical spring which operates with a coiling or uncoiling action as a door spring.



Spring, Valve - A helical spring used on valve stems and similar places; each coil being smaller than the one below, in order that the spring may close up into a very small space and then have a considerable range of action.

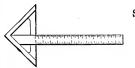
Spring Cotter - See Cotter.

Sprue Cutter — A cutting punch for trimming sprues from soft metal

Square, Caliper — A square with a caliper adjustment for laying out work.

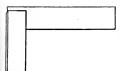


Square, Combination - A tool combining square, level and protractor in one tool.



Square, Center - For finding the center of a round bar by placing across the end and scribing lines in two different positions. Also used as a T-square. Not so much used as formerly.

T-Square — A straight edge with a head at one end commonly used on the drawing board for drawing straight lines. It forms a guide also, along which triangles are slid. Generally made of wood, although sometimes of metal and often provided with a swiveling head which serves as a protractor when graduated in degrees.



Square, Try - Small square for testing work as to its being at right angles.

Stand, Vise - Stand, usually of metal, for holding a vise firmly in any desired part of the shop, making it a portable vise.

Steady Rest - A rest attached to the lathe ways for supporting long. slender work.

Steel, High Speed - A name given to steels which do not lose their hardness by being heated under high speed cuts. Alloy steels which depend on tungsten, chromium, manganese, molybdenum. etc., for their hardness.

Stocks, Ratchet - Die stocks with ratchet handles.

scraped flat and true.

Straight Edge - A piece of metal having one edge ground and Small ones are sometimes made of steel but large, straight edges are usually of cast iron, proportioned to resist bending, and are used for testing the truth of flat

surfaces such as plane ways.

Strap — See Belt Polisher.

Strapping — A method of buffing by the use of a flexible strap or belt, usually made of cloth and covered with abrasive held in place by glue. Runs over two pulleys or one pulley and a rod or plate at high speed.

String Jig - Fixture for holding a row of pieces to be milled or planed. Stripper - A thin plate placed over the die, in a punch press, with

a gap beneath to admit the sheet stock and an opening to allow the punch to pass freely; upon the up-stroke of the punch it prevents the strip of metal from lifting with the punch.

Stripping-plate - A plate containing holes of the same outline as the pattern and used to prevent sand following the patterns when

drawn out on some molding machines.

Stud, Collar — The collar stud forms a satisfactory device for carrying gears, cam rolls, rocker levers, etc.

It is often provided with a hole at the end for a cutter pin or is slotted for a split washer, to retain the gear, or other

part in place.

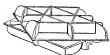
Stud, Shoulder — A stud of this form is used for mounting levers and

other parts which could be operated on a plain, unthreaded stud, which stud, however, cannot be conveniently set or removed when necessary. It is also a form of post or guide some-

times employed in machine construction for carrying one or more sliding parts.

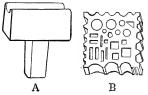
Stud, Threaded - Studs are threaded on both ends to lengths required and screwed tight into place. A nut is run on the outer

end. They are commonly used for holding cylinder heads in place and for other purposes where it is desirable that the screw shall remain stationary to prevent injury of threads tapped in the main piece.

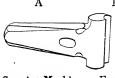


Surface Plates - Cast-iron plates have surfaces scraped flat for use in testing work. Should be made in sets of three and so scraped that each one has a perfect bearing with the other two.

Swaging - Changing the sectional shape of a piece of metal by hammering, rolling or otherwise forcing the particles to change shape without cutting.



Swaging Blocks - Blocks of cast or wrought iron to assist blacksmith in swaging and bending iron to various shapes. A is for use in the hardy hole in the anvil, B can be used anywhere but is usually on or beside the anvil.



Swaging Hammer - A connection with the swaging block to swage metal to the desired size and shape.

Swaging Machine - For reducing tapering or pointing wire or tubing either between rolling dies or by hammering with rapid blows between dies of suitable shape.

Sweating — Another name for soldering.

Swing of a Lathe - In the United States the swing of a lathe means the largest diameter of the work that can be swung over the ways or shears. In England it means the distance from lathe center to the ways or one half the U.S. measurements.

т

Take-up - Name given to device for taking up slack in belt or rope

Tap — Hardened and tempered steel tool for cutting internal threads. Has a thread cut on it and flutes to give cutting edges.



Tap, Bit-brace — Tap of any kind, usually on all bolt taps, with shank made square to be driven by bit-brace.

Tap, Echols Thread — This form of tap has every other thread cut away on each land, but these are staggered so that a space on one land has a tooth behind it on the next land. This is done for chip clearance.

Tap, Hand, First or Taper — Bolt tap usually for hand use. The first or taper tap has the front end tapered

to enter easily.







cutting threads in pipe dies. Taper inch per foot.

Tap, Hob for Solid Dies — Used for cutting the threads in a solid

Tap, Hob for Solid Dies — Used for cutting the threads in a solid die. It is best to remove about three fourths of the stock with a leading tap but is not necessary.

Tap, Hob, Sellers — Has threads in center and numerous flutes. For hobbing dies and chasers.

Tap, Machine or Nut— Tap with long shank small enough to allow tapped nuts to pass over it. After tap is full the tap is removed from machine and nuts slid off the shank.

Tap, Machine Screw — Taps made with sizes and threads of machine screws.

Made with shank the size of screw and pointed ends on small sizes.

Tap, Master - Tap for cutting solid and open dies.

Tap, Patch-bolt — Tap for boiler-makers use in patching boilers. Sizes vary by sixteenths from ½ inch to 1½ inches. All threads are 12 to inch and taper is ½ inch per foot.

Tap, Pipe — Taper tap, 3 inch taper per foot for pipe fitting.

Tap, Pulley — Tap with a long shank to reach the hub of pulley for tapping set-screw holes.

Tap Remover — Device for removing broken taps. Usually have prongs which go down in the flutes and around the central portion.

Tap, Staybolt — Tap for threading boiler sheets for staybolts. A reams the hole, B is a taper thread, C is straight thread of right size. D square for driving tap. All standard staybolt taps have 12 threads per inch.

Tap, Step — Tap made with "steps" or varying diameters. Front end cuts part of thread, next step takes out more and so on to the end. Only used for heavy threads, usually square or Acme.

Tap, Stove-bolt — Made same way as machine-screw taps but in only six standard sizes.

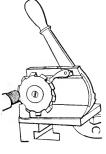
Tap, Tapper — Similar to a machine tap except that it has no square on the end.

Taper Reamer - See Reamer, Taper.

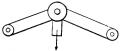
Tapped Face-plate — Having a number of tapped holes instead of slots. Studs screw in at any desired point.

Tapping Machine — For cutting threads with taps (tapping) in nuts or other holes.

Threads, Screw - See Screw Threads.



Threading Tool, Rivet-Dock — The tool is a rotary cutter with cutting teeth of different depths. The first tooth starts the cut, then instead of feeding the carriage into the work, the cutter is turned and the next tooth takes the next cut.



Toggle — Arrangement of levers to multiply pressure obtained by making movement given to work very much less than movement of applied power.

Tongs — Tools for holding hot or cold metals.

Tool, Boring — For operating on internal surface of holes.

Tool, Cutting-off — For cutting work apart on lathe or cutting-off machine.

Tool, Diamond — Black diamond set in metal for tracing emery or other abrasive wheels. Also used to some extent for truing up hardened steel or iron.

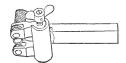
Tool Holder, Lathe or Planer — A body or shank, adopted to hold small pieces of tool steel for cutting tools



small pieces of tool steel for cutting tools. These can be removed for sharpening or renewal without moving the holder and saves resetting the tool to the work.

Tools, Inserted Cutter — Holders in which are held small steel cutting tools. These are usually removed for grinding or replacing when broken or worn out. Usually made of self-hardening steel.

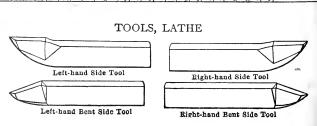
Tool, Nurling — For roughing or checking the outside of turned work so it can be readily grasped by hand. The tool is a wheel

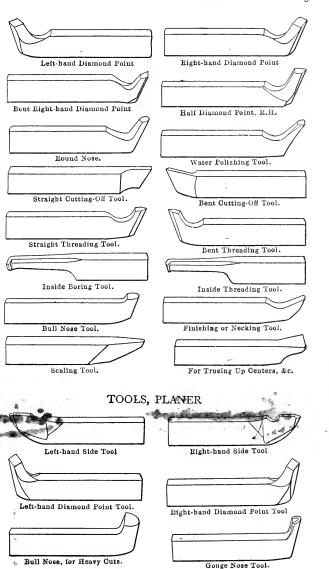


with the desired markings cut in the edge and hardened. It is forced against the work and actually forces the metal up into the depressions in the wheel. Most nurls are held in the end of a hand tool but for heavy nurling they are made to go in the tool post as shown.

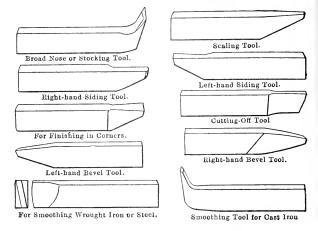
TOOLS, LATHE -- WM. SELLERS & CO.

		Lat	he		-	
Square Thread	60° V Thread	Bent Side	Side on an an an an an an an an an an an an an	Bent Roughing	Roughing and Cd.	Kind of Tool
Side a 1 Side b 1 End o 1	Side a 12 Side b 7 End c 15 Top d 0	Side a 6 End c 6 Top d 12	Side a 6 End c 6 Top d 12	Side a 5 Side b Side c 6 Top d 22	Side a 5 Side b Side o 6 Top d 22	Face
Square Thread	60° V Thread	Bent Side	Side	Nicking	Finishing	Clearance
		St. C.	C1 0 8	Niceting	Finishing	Kind of Tool
Left	Left	Left	Left	3 1 2 2 3 2 3 2 3	-1::1::1::2	
Side a Side b End c Top d	Side a Side b End c Top d	Side a End o Top d	Side a Erd c Top d	Side a Side b End o Top d	Side a Side b End o Top d	Face
Square Thread	o 5 5 -	10,000	12, 6, 6,	- ೧ ಬ ಬ ಬ	50 00 + 6	Clearance
Square Thread Bent	Bent Bent	Inside Bent	Bent Brass	Bent Nicking	Bent Finishing	Kind of Tool
Right	Right	Left	Right	Left	Left	
Side b End o Top d	Side a Side b End c	Side a End o Top d	Side a Side b End o Top d	Side a Side b End o Top d	Side a Side b End o Top d	Face
0 0 0 5	0, 5, 7, 5,	10, 0, 0,	0, 10, 6, 10,	⊢್ರ ೧ _೦ ಬ _{್ರ} ಬ್ರ	50 00 to 40	Clearance
Square Thread Bent	60° V Thread ⊕ Bent	Inside Bent	Brass	Bent.Nicking	Bent Finishing	Kind
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Left .	Left	Right	War Com.	Right	Right	78
Side a Side b End o	Side to Side to End o	20 2	Side a Side b End c	Side a Side b End c Top d	Side a Side b End o	Face
ල _ා ල _ා ල _ා	0, 5, 5, 1,	15° 0° 0	0 0 0 0 0	 요 행 영	5° ° 4° 4°	Clearance





Slotter



TOOLS, PLANER AND SLOTTER

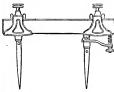
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Tote Boxes - See Tote Pans.

Tote Pans — Pans or trays of steel for carrying small parts from one part of shop to another.

Train — A series of gears, as the feed train of a lathe connecting spindle with lead screw.



Trammels — For drawing large circles.

Fit on a beam and their capacity depends on the length of the beam.

Trepanning Tool — Tool for cutting an annular groove outside or around a bored hole.

Tripper — Device that trips any piece of mechanism at the desired time. An example is found in conveyers where the tripper dumps the material at the desired point.

Tumbler Gear — An intermediate gear which meshes in between other gears to reverse the direction of the driven gear of the train.



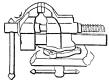
Turn Buckle — Turn buckles are for connecting and tightening truss rods, tie rods, etc., used in construction work.

Hook and Eye

Tuyere — The pipe or opening into forge through which air is forced.

V

Veeder Metal — An alloy with tin as a base, used for casting in metal molds.



Vise, Chipping or Filing — Heavy bench vise used for holding work to chip. Vises for filing only are similar but lighter.

Vise, Drill — Vise for use on drill press to hold work being drilled.



Vise, Hand—A small vise to be held in the hand. For small work that requires turning frequently to get at different sides.

Vise, Jig — A drill vise with arms which carry bushings so that pieces can be drilled in duplicate without special jigs for them.

Vise, Pin — Small hand vise for holding small wire rods.

Vise Stands - See Stands, Vise.

V's — Ways shaped like a V, either raised above the bed as on a lathe or cut below as in a planer, for guiding the travel of a carriage or table.

Volt - The unit of electrical pressure.

W

Washer, Open — Washers with one side open so as to be removed or slipped under the nut to avoid necessity of taking the nut entirely off. Also called a "C" washer.

Watt — The unit of electrical power and equals volts multiplied by amperes. 746 watts are equal to one horse-power.

Ways — The guiding or bearing surfaces on which moving parts slide, as in a lathe plane or milling machine. The ways may be of any form, flat, V or any other shape.

Welding — The joining of metals by heating the parts to be joined to the fusing point and making a union by hammering or forcing them together. Welding in an open fire is usually confined to iron and steel but nearly all metals can be joined in this way by electric heating.

. Wind — Pronounced with a long i as in "mind" and refers to a twist or warping away from straightness and parallelism.

Wrench, Bridge Builders' — Large heavy wrench with a hole in end for a tackle to apply power.

WRENCHES, MACHINE

15 degree angle wrenches have an opening milled at an angle of 15 degrees with the handle, which permits the turning of a hexagon nut completely around where the swing of the handle is limited to 30 degrees.

22½ degree angle wrenches have an opening which forms an angle of 22½ degrees with the handle, which permits the turning of any square head bolt or screw completely around where the swing of the

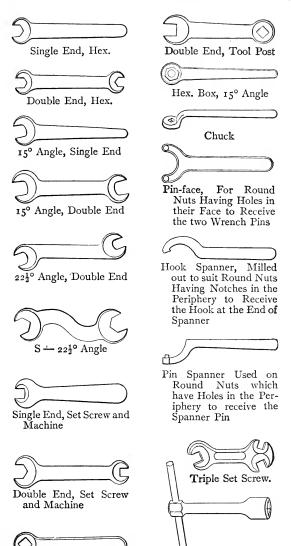
handle is limited to 45 degrees.

Unfinished drop-forged wrenches are plain forgings, with openings milled to fit the nut or screw on which they are to be used.

Semi-finished wrenches are milled to fit the nut or screw on which

they are to be used and case-hardened all over.

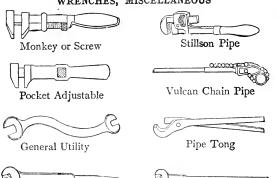
Finished wrenches are milled to fit the nut or screw on which they are to be used and are ground, polished, case-hardened all over, lacquered, with heads bright.



Socket

"Box"-Tool Post

WRENCHES, MISCELLANEOUS





Construction

Wrench, Tap — Wrench for holding and turning taps. Usually made adjustable for different sizes.

Track

Wringing Fits -- See Fits.

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