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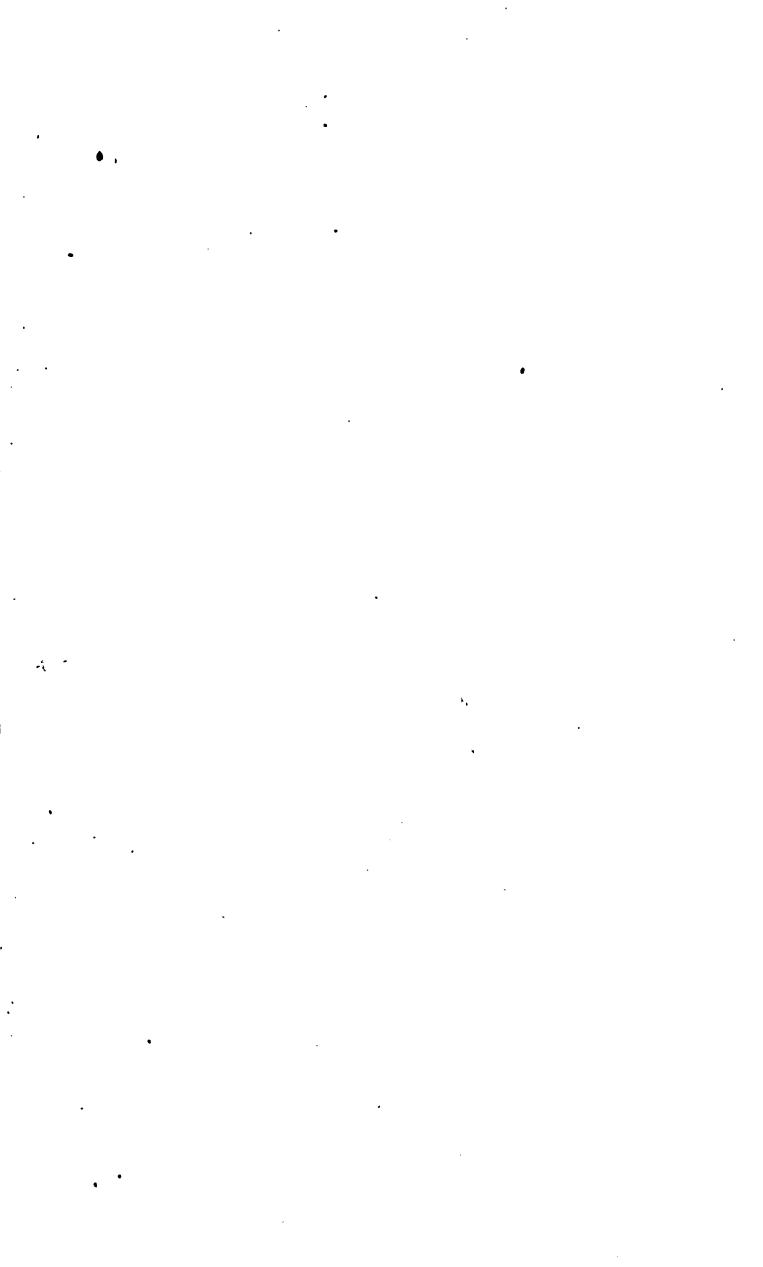
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HANDBOOK
OF
ENGINEERING MATHEMATICS

BY

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AUTHORS' PREFACE

IN the present edition, the handbook has been revised to include a number of additions to the mathematical sections and to the tables of mathematical functions, and the values of physical and chemical constants have been revised to agree with recent investigation.

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NOVEMBER, 1919.

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TABLE OF CONTENTS

| | PAGE |
|--|---------------|
| ALGEBRA | I |
| Exponents..... | I |
| Binomial theorem..... | 2 |
| Proportion..... | 2 |
| Progressions..... | 3 |
| Logarithms..... | 4 |
| Series..... | 6 |
| Complex imaginary quantities..... | 9 |
| GEOMETRY | 12 |
| Plane figures..... | 12 |
| Solids..... | 14 |
| PLANE TRIGONOMETRY | 15 |
| Numerical values..... | 16 |
| Trigonometric formulæ..... | 17 |
| Plane triangles..... | 19 |
| SPHERICAL TRIGONOMETRY | 21 |
| Formulæ..... | 21 |
| Application to navigation..... | 22 |
| PLANE ANALYTIC GEOMETRY | 24 |
| Straight line..... | 24 |
| Rectangular and polar coördinates..... | 26 |
| Circle..... | 26 |
| Parabola..... | 27 |
| Ellipse..... | 29 |
| Hyperbola..... | 29 |



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Special and Indeterminate Forms

$$a^0 = 1$$

$$a^\infty = \infty, \quad a > 1$$

$$a^{-\infty} = \frac{1}{a^\infty} = \frac{1}{\infty} = 0, \quad a > 1$$

$$\frac{a}{0} = \infty \quad \frac{a}{\infty} = 0$$

$$\frac{\infty}{a} = \infty \quad \frac{0}{a} = 0$$

$0 \cdot \infty$, $\frac{0}{0}$, $\frac{\infty}{\infty}$, 0^0 , 1^∞ , ∞^0 , $\infty - \infty$ are indeterminate.

For the evaluation of indeterminate forms, see page 50.

Binomial Theorem

$$(x + y)^n = x^n + nx^{n-1}y + \frac{n(n-1)}{2!}x^{n-2}y^2 + \frac{n(n-1)(n-2)}{3!}x^{n-3}y^3 + \dots$$

$$(1+x)^n = 1 + nx + \frac{n(n-1)}{2!}x^2 + \frac{n(n-1)(n-2)}{3!}x^3 + \dots$$

Proportion

If $a : b = c : d$ or $\frac{a}{b} = \frac{c}{d}$

then

$$ad = bc \quad \frac{a}{c} = \frac{b}{d} = \frac{a+b}{c+d} = \frac{a-b}{c-d}$$

$$\frac{b}{a} = \frac{d}{c} = \frac{b+d}{a+c} = \frac{b-d}{a-c}$$

If $\frac{a}{b} = \frac{c}{d}$ and $\frac{e}{f} = \frac{g}{h}$

then

$$\frac{ae}{bf} = \frac{cg}{dh} \quad \text{and} \quad \frac{ag}{bh} = \frac{ce}{df}$$

If

$$\frac{a}{b} = \frac{c}{d} = \frac{e}{f}$$

then

$$\frac{a + c + e}{b + d + f} = \frac{ma + nc + pe}{mb + nd + pf} = \frac{a}{b} = \frac{c}{d} = \frac{e}{f}$$

Arithmetical Progression

An **arithmetical progression** is one whose terms increase or decrease by a common difference,

$$a, a + d, a + 2d, a + 3d, \dots$$

the last term is $L = a + (n - 1)d$

the sum of the terms is

$$S = \frac{n}{2}(a + L) = \frac{n}{2}[2a + (n - 1)d]$$

a = first term

n = number of terms

d = common difference

Geometrical Progression

Quantities are in **geometrical progression** when each term is equal to the preceding term multiplied by a constant,

$$a, ar, ar^2, ar^3, \dots$$

the last term is $L = ar^{n-1}$

the sum of the terms is

$$S = \frac{a(r^n - 1)}{r - 1} = \frac{a(1 - r^n)}{1 - r} = \frac{rL - a}{r - 1}$$

a = first term

r = constant ratio

n = number of terms

The **sum** of an **infinite number** of terms in geometrical progression is

$$S = \frac{a}{1 - r}$$

in which the ratio r must be less than 1 if the series is to be convergent (see Infinite Series).

Logarithms

The **logarithm** of any number to a given base is the power to which the base must be raised in order to produce the given number, thus:

$$\text{if } x^m = y, \text{ then } m = \log_x y,$$

that is, m is the logarithm of y to the base x .

The following relations hold for any base:

$$\log ab = \log a + \log b$$

$$\log \frac{a}{b} = \log a - \log b$$

$$\log a^n = n \log a$$

$$\log \frac{1}{a} = -\log a$$

The **base of the common system** of logarithms is 10.

The **base of the natural system** of logarithms (also called Naperian or hyperbolic logarithms) is $e = 2.7182818284 \dots$

A logarithm may be transformed from any given base to any other desired base by the relation:

$$\log_b N = \frac{\log_a N}{\log_a b}$$

To transform a logarithm from base 10 to base e ,

multiply by 2.302585 . . . (where 2.302585 . . . is the logarithm of 10 to the base e):

$$\log_e a = 2.302585 \log_{10} a$$

To transform a logarithm from base e to base 10, divide by 2.302585:

$$\log_{10} a = \frac{1}{2.302585} \log_e a = 0.434294 \log_e a$$

Special forms:

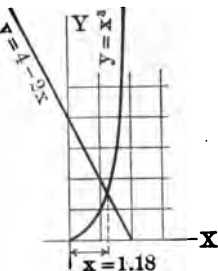
$$\begin{aligned} \log 1 &= 0 && \text{(to any base)} \\ \log_a a &= 1 && \log_e e = 1 \\ \log_b 0 &= -\infty && \log_b \infty = \infty \\ &&& b > 1 \end{aligned}$$

Cubic and Higher Degree Equations

The approximate values of the real roots of an algebraic equation containing only one variable may be found graphically.

For instance, let it be required to solve the equation $x^3 + Ax - B = 0$. This may be written as $x^3 = -Ax + B$, or as two simultaneous equations $y = x^3$ and $y = -Ax + B$. The graph of each of these equations being plotted, the abscissas of their points of intersection give the real roots of the cubic. The curve $y = x^3$ should be plotted on cross-section paper by the aid of a table of cubes. The curve $y = -Ax + B$ is the equation of a straight line, and is therefore determined by plotting two points.

Illustrative Example. Solve the equation $x^3 + 2x - 4 = 0$ graphically. Write the equation in the form $x^3 = 4 - 2x = y$ and plot the equations $y = x^3$ and $y = 4 - 2x$. Their intersection gives the solution $x = 1.18$.



Algebraic equations of any degree may be solved by Newton's method of approximation; see page 51.

Transcendental Equations

The graphic method given under Cubic and Higher Degree Equations is also applicable to many transcendental equations. Thus, the equation $Ax - \sin x = 0$ may be solved by plotting the two simultaneous equations $y = Ax$ and $y = \sin x$. The curve $y = \sin x$ is readily plotted with the aid of a table of sines, while the other curve $y = Ax$ is a straight line passing through the origin.

Infinite Series

An **infinite series** is one containing an unlimited number of terms. Such a series is **convergent** if the sum of its terms is a finite quantity. It is **divergent** when the sum of its terms does not approach a finite limit.

Comparison Test. A series is converging if each term in it is equal to or less than the corresponding term of a known converging series.

Converging series for comparison:

$$a + ar + ar^2 + ar^3 + \dots + ar^{n-1} + \dots \quad [r < 1]$$

$$1 + \frac{1}{2} + \frac{1}{4} + \frac{1}{8} + \dots + \frac{1}{2^{n-1}} + \dots$$

$$\frac{1}{1 \cdot 2} + \frac{1}{2 \cdot 3} + \frac{1}{3 \cdot 4} + \dots + \frac{1}{n(n+1)} + \dots$$

$$1 + \frac{1}{2^p} + \frac{1}{3^p} + \dots + \frac{1}{n^p} + \dots \quad [p > 1]$$

A series is diverging if each term in it is equal to or greater than the corresponding term of a known diverging series.

Diverging series for comparison:

$$a + ar + ar^2 + ar^3 + \dots + ar^{n-1} + \dots \quad [r \geq 1]$$

$$1 + 1 + 1 + 1 + 1 + \dots$$

$$1 + \frac{1}{2} + \frac{1}{3} + \frac{1}{4} + \dots + \frac{1}{n} + \dots$$

Ratio Test. If, as the number of terms approaches infinity as its limit, the ratio of the $(n + 1)$ th term to the n th term approaches some finite limit (a), the series is convergent if (a) is less than 1, divergent if (a) is greater than 1, and indeterminate by this method if (a) = 1.

Oscillating Series. A series whose terms are alternately positive and negative is convergent if each term is numerically less than the preceding term.

Standard Series

$$e^x = 1 + x + \frac{x^2}{2!} + \frac{x^3}{3!} + \frac{x^4}{4!} + \dots$$

$$e^{-x} = 1 - x + \frac{x^2}{2!} - \frac{x^3}{3!} + \frac{x^4}{4!} - \dots$$

$$e^{jx} = e^{\sqrt{-1}x} = 1 + jx - \frac{x^2}{2!} - \frac{jx^3}{3!} + \frac{x^4}{4!} + \dots$$

$$e^{-jx} = e^{-\sqrt{-1}x} = 1 - jx - \frac{x^2}{2!} + \frac{jx^3}{3!} + \frac{x^4}{4!} - \dots$$

$$e = 1 + \frac{1}{1!} + \frac{1}{2!} + \frac{1}{3!} + \dots = \lim_{n \rightarrow \infty} \left(1 + \frac{1}{n}\right)^n \\ = 2.7182818 \dots$$

$$a^x = 1 + x \log a + \frac{(x \log a)^2}{2!} + \frac{(x \log a)^3}{3!} + \dots$$

$$\log x = 2 \left[\frac{x-1}{x+1} + \frac{1}{3} \left(\frac{x-1}{x+1} \right)^3 + \frac{1}{5} \left(\frac{x-1}{x+1} \right)^5 + \dots \right] \quad [x > 0]$$

$$\log x = \frac{x-1}{x} + \frac{1}{2} \left(\frac{x-1}{x} \right)^2 + \frac{1}{3} \left(\frac{x-1}{x} \right)^3 + \dots \quad [x > \frac{1}{2}]$$

$$\log x = (x-1) - \frac{1}{2}(x-1)^2 + \frac{1}{3}(x-1)^3 - \dots \quad [2 > x > 0]$$

$$(1 \pm x)^{-1} = 1 \mp x + x^2 \mp x^3 + x^4 \mp x^5 + \dots \quad [x^2 < 1]$$

$$\log(1+x) = x - \frac{x^2}{2} + \frac{x^3}{3} - \frac{x^4}{4} + \dots \quad [1 \geq x > -1]$$

$$\log(1-x) = -x - \frac{x^2}{2} - \frac{x^3}{3} - \frac{x^4}{4} - \dots \quad [1 > x \geq -1]$$

$$\sin x = x - \frac{x^3}{3!} + \frac{x^5}{5!} - \frac{x^7}{7!} + \frac{x^9}{9!} - \dots$$

$$\cos x = 1 - \frac{x^2}{2!} + \frac{x^4}{4!} - \frac{x^6}{6!} + \frac{x^8}{8!} - \dots$$

$$\tan x = x + \frac{x^3}{3} + \frac{2x^5}{15} + \frac{17x^7}{315} + \frac{62x^9}{2835} + \dots$$

$$\left[\frac{\pi}{2} > x > -\frac{\pi}{2} \right]$$

$$\cot x = \frac{1}{x} - \frac{x}{3} - \frac{x^3}{45} - \frac{2x^5}{945} - \frac{x^7}{4725} - \dots \quad [x^2 < \pi^2]$$

$$\sin^{-1} x = x + \frac{x^3}{6} + \frac{1}{2} \cdot \frac{3}{4} \cdot \frac{x^5}{5} + \frac{1}{2} \cdot \frac{3}{4} \cdot \frac{5}{6} \cdot \frac{x^7}{7} + \dots \quad [1 > x > -1]$$

$$\cos^{-1} x = \frac{\pi}{2} - \sin^{-1} x$$

$$\tan^{-1} x = x - \frac{x^3}{3} + \frac{x^5}{5} - \frac{x^7}{7} + \dots \quad [1 > x > -1]$$

$$\tan^{-1} x = \frac{\pi}{2} - \frac{1}{x} + \frac{1}{3x^3} - \frac{1}{5x^5} + \dots \quad [x^2 > 1]$$

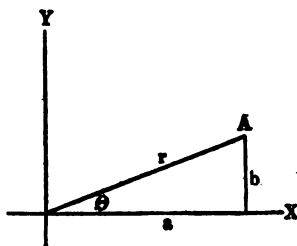
$$\sinh x = x + \frac{x^3}{3!} + \frac{x^5}{5!} + \frac{x^7}{7!} + \frac{x^9}{9!} + \dots$$

$$\cosh x = 1 + \frac{x^2}{2!} + \frac{x^4}{4!} + \frac{x^6}{6!} + \frac{x^8}{8!} + \dots$$

Complex Imaginary Quantities

The imaginary unit = $\sqrt{-1} = j$

In representing complex imaginary quantities, it is usual to represent real quantities in the direction of the horizontal or X -axis, and imaginaries in the direction of the vertical or Y -axis. Multipli-



cation by the imaginary unit, j , revolves a quantity through 90 degrees, in counter-clockwise direction.

A complex number is the sum of a real and an imaginary, thus:

$$A = a + jb = a + \sqrt{-1} b$$

is a complex number.

A complex number may be written in any of the following identical forms:

$$A = a + jb = r (\cos \theta + j \sin \theta) = re^{j\theta} \quad [\theta \text{ in radians}]$$

in which $\begin{cases} a = r \cos \theta, \\ b = r \sin \theta. \end{cases}$

The magnitude of the complex number, $a + jb$, is

$$r = \sqrt{a^2 + b^2}$$

Addition and Subtraction of complex quantities:

To add two complex quantities, combine the real parts, and then the imaginaries, thus:

$$(a + jb) + (c + jd) = (a + c) + j(b + d)$$

In the same way, to subtract two complex quantities:

$$(a + jb) - (c + jd) = (a - c) + j(b - d)$$

Multiplication of complex quantities:

To find the product of two complex numbers, multiply out as in ordinary algebra, remembering that $j^2 = -1$, thus:

$$(a + jb)(c + jd) = (ac - bd) + j(ad + bc)$$

Division of complex quantities:

To divide two complex quantities, rationalize the denominator as follows:

$$\frac{a + jb}{c + jd} = \frac{a + jb}{c + jd} \times \frac{c - jd}{c - jd} = \frac{(ac + bd) + j(bc - ad)}{c^2 + d^2}$$

Logarithms of complex quantities:

To obtain the logarithm of a complex quantity, use the following formulæ:

$$\begin{aligned} \log_e(a + jb) &= \log_e r(\cos \theta + j \sin \theta), \quad \text{where } r = \sqrt{a^2 + b^2} \\ &= \log_e (r e^{j\theta}) \\ &= \log_e r + \log_e e^{j\theta} \\ &= \log_e r + j\theta \end{aligned}$$

$$\log_n(a + jb) = \log_n (r e^{j\theta}) = \log_n r + j\theta \log_n e$$

Complex Imaginary Formulæ

$$j = \sqrt{-1}$$

$$j^2 = jj = -1$$

$$e = 2.71828 +$$

$$e^{jax} = \cos ax + j \sin ax = \cosh jax + \sinh jax$$

$$e^{-jax} = \cos ax - j \sin ax = \cosh jax - \sinh jax$$

$$e^{ax} = \cos jax - j \sin jax = \cosh ax + \sinh ax$$

$$e^{-ax} = \cos jax + j \sin jax = \cosh ax - \sinh ax$$

$$\sin ax = \frac{e^{jax} - e^{-jax}}{2j} = \frac{\sinh jax}{j}$$

$$\cos ax = \frac{e^{jax} + e^{-jax}}{2} = \cosh jax$$

$$\sin jax = j \frac{e^{ax} - e^{-ax}}{2} = j \sinh ax$$

$$\cos jax = \frac{e^{ax} + e^{-ax}}{2} = \cosh ax$$

$$e^{u \pm jv} = e^u (\cos v \pm j \sin v)$$

$$(\cos \theta + j \sin \theta)^n = \cos n\theta + j \sin n\theta$$

(De Moivre's theorem)

$$e^{j\frac{\pi}{2}} = \cos \frac{\pi}{2} + j \sin \frac{\pi}{2} = j$$

$$e^{-j\frac{\pi}{2}} = \cos \frac{\pi}{2} - j \sin \frac{\pi}{2} = -j$$

$$e^{j(\theta + \frac{\pi}{2})} = e^{j\theta} e^{j\frac{\pi}{2}} = je^{j\theta}$$

$$e^{j(\theta - \frac{\pi}{2})} = e^{j\theta} e^{-j\frac{\pi}{2}} = -je^{j\theta}$$

Permutations and Combinations

The number of permutations of n different things taken r at a time is

$$P_r = n(n-1) \dots (n-r+1) = \frac{n!}{(n-r)!}$$

For n different things taken all at a time, the number of permutations is

$$P_n = n(n-1) \dots (2)(1) = n!$$

The number of permutations of n things taken all at a time, n_1 being alike, n_2 alike, n_3 alike, etc., is

$$P = \frac{n!}{n_1! n_2! n_3!} \dots$$

The number of combinations of n things taken r at a time is

$$C_r = \frac{n(n-1) \dots (n-r+1)}{r!} = \frac{n!}{r!(n-r)!}$$

For n things taken 1, 2, 3, . . . n at a time, the total number of combinations is

$$C = 2^n - 1$$

GEOMETRY

Plane Figures

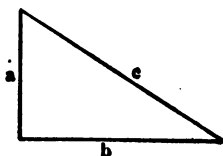
Right Triangle

$$c = \sqrt{a^2 + b^2}$$

$$a = \sqrt{c^2 - b^2}$$

$$b = \sqrt{c^2 - a^2}$$

$$\text{area} = \frac{1}{2} ab$$

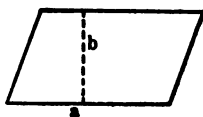
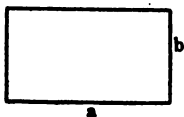
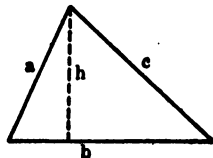


Any Triangle

$$\text{area} = \frac{1}{2} bh$$

$$\text{area} = \sqrt{s(s-a)(s-b)(s-c)}$$

$$s = \frac{1}{2}(a+b+c)$$

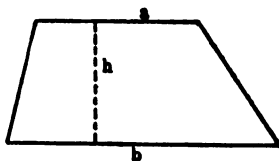


Parallelogram

$$\text{area} = ab$$

Trapezoid

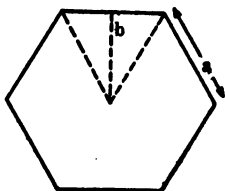
$$\text{area} = \frac{1}{2} h(a+b)$$



Regular Polygon

$$\text{area} = \frac{1}{2} abn$$

n = number of sides



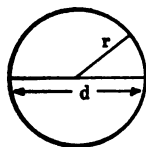
Circle

$$\text{circumference} = 2\pi r$$

$$= \pi d$$

$$\text{area} = \pi r^2$$

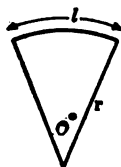
$$= \pi \frac{d^2}{4}$$



Sector of Circle

$$\text{arc} = l = \pi r \frac{\theta^\circ}{180^\circ}$$

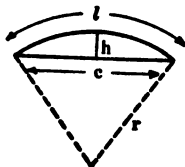
$$\text{area} = \frac{1}{2} rl = \pi r^2 \frac{\theta^\circ}{360^\circ}$$



Segment of Circle

$$\text{chord} = c = 2\sqrt{2hr - h^2}$$

$$\text{area} = \frac{1}{2} rl - \frac{1}{2} c(r - h)$$



Parabola

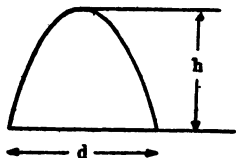
$$\text{length of arc} = \frac{d^2}{8h} [\sqrt{c(1+c)} +$$

$$2.0326 \log_{10} (\sqrt{c} + \sqrt{1+c})]$$

in which

$$c = \left(\frac{4h}{d}\right)^2$$

$$\text{area} = \frac{2}{3} dh$$



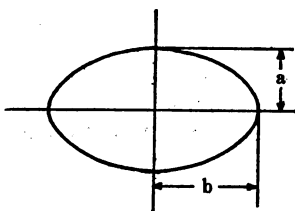
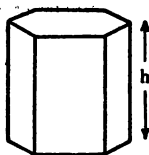
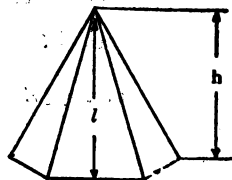
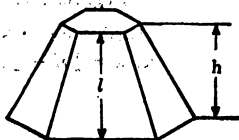
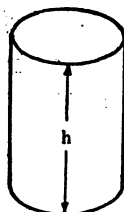
Ellipse

circumference =

$$\pi(a+b) \frac{64 - 3\left(\frac{b-a}{b+a}\right)^4}{64 - 16\left(\frac{b-a}{b+a}\right)^2}$$

(close approximation)

$$\text{area} = \pi ab$$

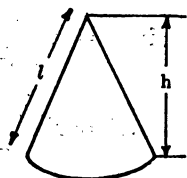
**Solids****Right Prism**lateral surface = perimeter of base \times h volume = area of base \times h **Pyramid**lateral area = $\frac{1}{2}$ perimeter of base \times l volume = area of base \times $\frac{h}{3}$ **Frustum of Pyramid**lateral surface = $\frac{1}{2} l (P + p)$ P = perimeter of lower base p = perimeter of upper basevolume = $\frac{1}{3} h [A + a + \sqrt{Aa}]$ A = area of lower base a = area of upper base**Right Circular Cylinder**lateral surface = $2\pi r h$ r = radius of basevolume = $\pi r^2 h$ 

Right Circular Cone

lateral surface = $\pi r l$

 r = radius of base

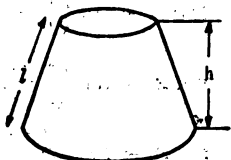
volume = $\frac{1}{3} \pi r^2 h$

**Frustum of Right Circular Cone**

lateral surface = $\pi l (R + r)$

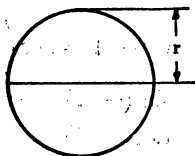
 R = radius of lower base r = radius of upper base

volume = $\frac{1}{3} \pi h [R^2 + Rr + r^2]$

**Sphere**

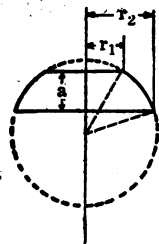
surface = $4 \pi r^2$

volume = $\frac{4}{3} \pi r^3$

**Segment of Sphere**

volume of segment

= $\frac{1}{6} a \pi [3(r_1^2 + r_2^2) + a^2]$

**PLANE TRIGONOMETRY****Right Triangle**

$\sin A = \frac{a}{c}$

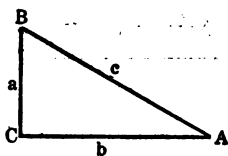
$\cos A = \frac{b}{c}$

$\tan A = \frac{a}{b}$

$\cot A = \frac{b}{a}$

$\sec A = \frac{c}{b}$

$\operatorname{cosec} A = \frac{c}{a}$



$$\sin A = \cos \left(\frac{\pi}{2} - A \right) = -\cos \left(\frac{\pi}{2} + A \right)$$

$$\cos A = \sin \left(\frac{\pi}{2} - A \right) = \sin \left(\frac{\pi}{2} + A \right)$$

$$\tan A = \cot \left(\frac{\pi}{2} - A \right) = -\cot \left(\frac{\pi}{2} + A \right)$$

$$\cot A = \tan \left(\frac{\pi}{2} - A \right) = -\tan \left(\frac{\pi}{2} + A \right)$$

$$\sec A = \operatorname{cosec} \left(\frac{\pi}{2} - A \right) = \operatorname{cosec} \left(\frac{\pi}{2} + A \right)$$

$$\operatorname{cosec} A = \sec \left(\frac{\pi}{2} - A \right) = -\sec \left(\frac{\pi}{2} + A \right)$$

$$\sin(-A) = -\sin A$$

$$\cos(-A) = \cos A$$

$$\tan(-A) = -\tan A$$

$$\cot(-A) = -\cot A$$

$$\sec(-A) = \sec A$$

$$\operatorname{cosec}(-A) = -\operatorname{cosec} A$$

NUMERICAL VALUES

| Angle .. | 0° | 30° | 45° | 60° | 90° |
|----------|----------|----------------------|----------------------|----------------------|----------|
| sin..... | 0 | $\frac{1}{2}$ | $\frac{\sqrt{2}}{2}$ | $\frac{\sqrt{3}}{2}$ | 1 |
| cos..... | 1 | $\frac{\sqrt{3}}{2}$ | $\frac{\sqrt{2}}{2}$ | $\frac{1}{2}$ | 0 |
| tan..... | 0 | $\frac{\sqrt{3}}{3}$ | 1 | $\sqrt{3}$ | ∞ |
| cot..... | ∞ | $\sqrt{3}$ | 1 | $\frac{\sqrt{3}}{3}$ | 0 |

SIGNS OF THE FUNCTIONS

| | sin | cos | tan | cot | sec | cosec |
|-------------------|-----|-----|-----|-----|-----|-------|
| 1st Quadrant..... | + | + | + | + | + | + |
| 2nd Quadrant..... | + | - | - | - | - | + |
| 3rd Quadrant..... | - | - | + | + | - | - |
| 4th Quadrant..... | - | + | - | - | + | - |

Trigonometric Formulæ

$$\tan x = \frac{\sin x}{\cos x} \quad \cot x = \frac{\cos x}{\sin x}$$

$$\sec x = \frac{1}{\cos x} \quad \operatorname{cosec} x = \frac{1}{\sin x}$$

$$\tan x = \frac{1}{\cot x} \quad \cot x = \frac{1}{\tan x}$$

$$\sin^2 x + \cos^2 x = 1$$

$$\sec^2 x = 1 + \tan^2 x$$

$$\operatorname{cosec}^2 x = 1 + \cot^2 x$$

$$\sin(x + y) = \sin x \cos y + \cos x \sin y$$

$$\cos(x + y) = \cos x \cos y - \sin x \sin y$$

$$\tan(x + y) = \frac{\tan x + \tan y}{1 - \tan x \tan y}$$

$$\cot(x + y) = \frac{\cot x \cot y - 1}{\cot x + \cot y}$$

$$\sin(x - y) = \sin x \cos y - \cos x \sin y$$

$$\cos(x - y) = \cos x \cos y + \sin x \sin y$$

$$\tan(x - y) = \frac{\tan x - \tan y}{1 + \tan x \tan y}$$

$$\cot(x - y) = \frac{\cot x \cot y + 1}{\cot y - \cot x}$$

$$\sin 2x = 2 \sin x \cos x$$

$$\cos 2x = \cos^2 x - \sin^2 x$$

$$\tan 2x = \frac{2 \tan x}{1 - \tan^2 x}$$

$$\cot 2x = \frac{\cot^2 x - 1}{2 \cot x}$$

$$\sin \frac{1}{2}x = \sqrt{\frac{1 - \cos x}{2}}$$

$$\cos \frac{1}{2}x = \sqrt{\frac{1 + \cos x}{2}}$$

$$\tan \frac{1}{2}x = \frac{1 - \cos x}{\sin x}$$

$$\sin x + \sin y = 2 \sin \frac{1}{2}(x + y) \cos \frac{1}{2}(x - y)$$

$$\sin x - \sin y = 2 \cos \frac{1}{2}(x + y) \sin \frac{1}{2}(x - y)$$

$$\cos x + \cos y = 2 \cos \frac{1}{2}(x + y) \cos \frac{1}{2}(x - y)$$

$$\cos x - \cos y = -2 \sin \frac{1}{2}(x + y) \sin \frac{1}{2}(x - y)$$

$$\begin{aligned} \sin x &= \sqrt{1 - \cos^2 x} = \frac{1}{\operatorname{cosec} x} = \frac{\cos x}{\cot x} = \frac{\tan x}{\sec x} \\ &= \cos x \tan x = \frac{\tan x}{\sqrt{1 + \tan^2 x}} = \frac{1}{\sqrt{1 + \cot^2 x}} \\ &= \frac{\sqrt{\sec^2 x - 1}}{\sec x} = \frac{\sin 2x}{2 \cos x} = \sqrt{\frac{1}{2}(1 - \cos 2x)} \\ &= 2 \sin \frac{x}{2} \cos \frac{x}{2} \end{aligned}$$

$$\begin{aligned} \cos x &= \sqrt{1 - \sin^2 x} = \frac{1}{\sec x} = \frac{\sin x}{\tan x} = \frac{\cot x}{\operatorname{cosec} x} \\ &= \sin x \cot x = \frac{\cot x}{\sqrt{1 + \cot^2 x}} = \frac{1}{\sqrt{1 + \tan^2 x}} \\ &= \frac{\sqrt{\sec^2 x - 1}}{\operatorname{cosec} x} = \frac{\sin 2x}{2 \sin x} = \sqrt{\frac{1}{2}(1 + \cos 2x)} \\ &= \cos^2 \frac{x}{2} - \sin^2 \frac{x}{2} = 2 \cos^2 \frac{x}{2} - 1 = 1 - 2 \sin^2 \frac{x}{2} \end{aligned}$$

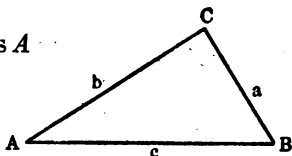
Equivalent expressions for $\tan x$ and $\cot x$ may be obtained by taking any of the above expressions for $\sin x$ and $\cos x$ and substituting in the equations

$$\tan x = \frac{\sin x}{\cos x} \qquad \cot x = \frac{\cos x}{\sin x}$$

Solution of Any Plane Triangle

I. Given any two sides b and c and their included angle A .

Use any one of the following sets of formulas: $\triangle ABC$



$$(1) \quad \frac{1}{2}(B + C) = 90^\circ - \frac{1}{2}A$$

$$\tan \frac{1}{2}(B - C)$$

$$= \frac{b - c}{b + c} \tan \frac{1}{2}(B + C)$$

$$B = \frac{1}{2}(B + C) + \frac{1}{2}(B - C)$$

$$C = \frac{1}{2}(B + C) - \frac{1}{2}(B - C)$$

$$a = \frac{b \sin A}{\sin B}$$

$$(2) \quad \tan C = \frac{c \sin A}{b - c \cos A}$$

$$B = 180^\circ - (A + C)$$

$$a = \frac{c \sin A}{\sin C}$$

$$(3) \quad a = \sqrt{b^2 + c^2 - 2bc \cos A}$$

$$\sin B = \frac{b \sin A}{a}$$

$$C = 180^\circ - (A + B)$$

II. Given any two angles A and B and any side c .

$$C = 180^\circ - (A + B)$$

$$a = \frac{c \sin A}{\sin C}$$

$$b = \frac{c \sin B}{\sin C}$$

III. Given the three sides a , b , and c . Use either of the following sets of formulas.

$$(1) \cos A = \frac{b^2 + c^2 - a^2}{2bc}$$

$$\cos B = \frac{a^2 + c^2 - b^2}{2ac}$$

$$C = 180^\circ - (A + B)$$

$$(2) \quad s = \frac{1}{2}(a + b + c)$$

$$r = \sqrt{\frac{(s-a)(s-b)(s-c)}{s}}$$

$$\tan \frac{1}{2} A = \frac{r}{s-a}$$

$$\tan \frac{1}{2} B = \frac{r}{s-b}$$

$$\tan \frac{1}{2} C = \frac{r}{s-c}$$

IV. Given any two sides a and b and an angle A opposite either one of these.

$$\sin C = \frac{c \sin A}{a}$$

$$B = 180^\circ - (A + C)$$

$$b = \frac{a \sin B}{\sin A}$$

NOTE. There may be two values for the angle C . If, however, one solution is such that $A + C > 180^\circ$, use other value only.

SPHERICAL TRIGONOMETRY

Right Spherical Triangles

$$\cos c = \cos a \cos b$$

$$\sin a = \sin c \sin A$$

$$\sin b = \sin c \sin B$$

$$\cos A = \cos a \sin B$$

$$\cos B = \cos b \sin A$$

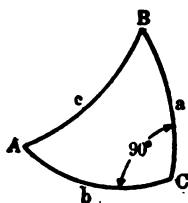
$$\cos A = \tan b \cot c$$

$$\cos B = \tan a \cot c$$

$$\sin b = \tan a \cot A$$

$$\sin a = \tan b \cot B$$

$$\cos c = \cot A \cot B$$



Oblique Spherical Triangles

$$\frac{\sin a}{\sin A} = \frac{\sin b}{\sin B} = \frac{\sin c}{\sin C}$$

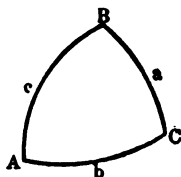
$$\cos a = \cos b \cos c + \sin b \sin c \cos A$$

$$\cos A = \sin B \sin C \cos a - \cos B \cos C$$

$$\cot a \sin b = \cot A \sin C + \cos C \cos b$$

$$s = \frac{1}{2} (a + b + c)$$

$$S = \frac{1}{2} (A + B + C)$$



$$\sin \left(\frac{A}{2} \right) = \sqrt{\frac{\sin (s-b) \sin (s-c)}{\sin b \sin c}}$$

$$\cos \left(\frac{A}{2} \right) = \sqrt{\frac{\sin s \sin (s-a)}{\sin b \sin c}}$$

$$\tan \left(\frac{A}{2} \right) = \sqrt{\frac{\sin (s-b) \sin (s-c)}{\sin s \sin (s-a)}}$$

$$\sin \left(\frac{a}{2} \right) = \sqrt{-\frac{\cos S \cos (S-A)}{\sin B \sin C}}$$

$$\cos \left(\frac{a}{2} \right) = \sqrt{\frac{\cos (S-B) \cos (S-C)}{\sin B \sin C}}$$

$$\tan \left(\frac{a}{2} \right) = \sqrt{-\frac{\cos S \cos (S-A)}{\cos (S-B) \cos (S-C)}}$$

$$\tan \frac{1}{2} (a-b) = \frac{\sin \frac{1}{2} (A-B)}{\sin \frac{1}{2} (A+B)} \tan \frac{1}{2} c$$

$$\tan \frac{1}{2} (a+b) = \frac{\cos \frac{1}{2} (A-B)}{\cos \frac{1}{2} (A+B)} \tan \frac{1}{2} c$$

$$\tan \frac{1}{2} (A-B) = \frac{\sin \frac{1}{2} (a-b)}{\sin \frac{1}{2} (a+b)} \cot \frac{1}{2} C$$

$$\tan \frac{1}{2} (A+B) = \frac{\cos \frac{1}{2} (a-b)}{\cos \frac{1}{2} (a+b)} \cot \frac{1}{2} C$$

$$\tan \frac{1}{2} c = \frac{\sin \frac{1}{2} (A+B) \tan \frac{1}{2} (a-b)}{\sin \frac{1}{2} (A-B)}$$

Application of Spherical Trigonometry to Navigation

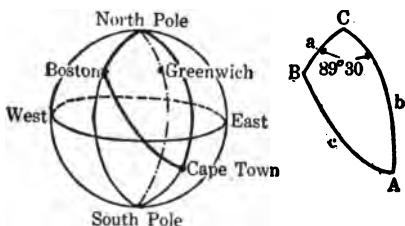
To find the shortest distance between two points on the earth's surface and the bearing of each from the other, the latitude and longitude of each being given. (From W. A. Granville's "Plane and Spherical Trigonometry.")

(1) Subtract the latitude of each place algebraically from 90° , taking North latitudes as positive and South latitudes as negative. The results will be the two sides of a spherical triangle.

(2) Find the difference of longitude of the two places by subtracting the lesser longitude from the greater if both are East or both are West; but adding the two if one is East and the other West. This gives the included angle of the triangle. If the difference of longitude found is greater than 180° , then subtract it from 360° and use the remainder as the included angle.

(3) Solving the triangle by the formulæ for $\tan \frac{1}{2}(A - B)$, $\tan \frac{1}{2}(A + B)$, and $\tan \frac{1}{2}c$, the third side gives the shortest distance between the two points in degrees of arc, and the angles give the bearings. The number of minutes in the arc will be the distance between the places in nautical miles.

Illustration. Find the shortest distance along the earth's surface between Boston (latitude $42^\circ 21' N.$,



longitude $71^\circ 4' W.$) and Capetown (latitude $33^\circ 56' S.$, longitude $18^\circ 26' E.$) and the bearing of each city from the other.

$$(1) \quad a = 90^\circ - 42^\circ 21' = 47^\circ 39'$$

$$b = 90^\circ - (-33^\circ 56') = 123^\circ 56'$$

- (2) $C = 71^\circ 4' + 18^\circ 26' = 89^\circ 30' =$ difference in longitude.
- (3) Solving the triangle as explained above, we get
 $c = 68^\circ 14' = 68.23^\circ = 4094$ nautical miles.
 $A = 52^\circ 43' =$ bearing of Boston from Capetown.
 $B = 116^\circ 43' =$ bearing of Capetown from Boston.

PLANE ANALYTIC GEOMETRY

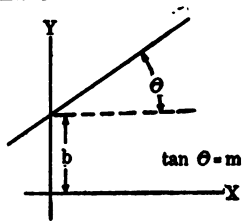
The Straight Line

I. The slope equation:

$$y = mx + b$$

$$m = \text{slope} = \tan \theta$$

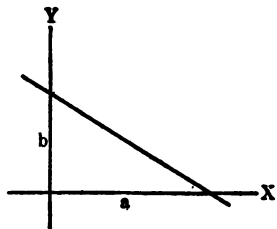
$$b = \text{intercept on } Y\text{-axis}$$



II. The intercept equation:

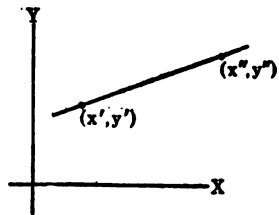
$$\frac{x}{a} + \frac{y}{b} = 1$$

where a and b are the intercepts on the X and Y -axes.



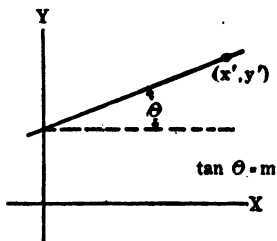
III. Line through the points (x', y') and (x'', y'') :

$$\frac{y - y'}{y'' - y'} = \frac{x - x'}{x'' - x'}$$



IV. Line through the point (x', y') , with slope m :

$$y - y' = m(x - x')$$



V. Distance from the point (x', y') to the line $Ax + By + C = 0$:

$$d = \frac{Ax' + By' + C}{\pm \sqrt{A^2 + B^2}}$$

VI. Distance between the points (x', y') and (x'', y'') :

$$d = \sqrt{(x' - x'')^2 + (y' - y'')^2}$$

VII. Area of a triangle with vertices at points (x_1, y_1) , (x_2, y_2) , and (x_3, y_3) .

$$A = \frac{1}{2}[x_1(y_2 - y_3) + x_2(y_3 - y_1) + x_3(y_1 - y_2)]$$

VIII. Angle between two lines with slopes m_1 and m_2 .

$$\tan \theta = \frac{m_2 - m_1}{1 + m_1 m_2}$$

NOTE. If $m_1 = m_2$ lines are parallel and if $m_1 = -\frac{1}{m_2}$ lines are perpendicular.

Transformation from Rectangular to Polar Coördinates

$$x = r \cos \theta$$

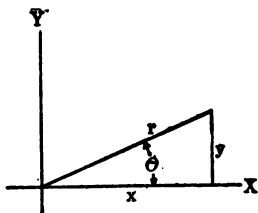
$$y = r \sin \theta$$

$$r = \text{radius vector} = \sqrt{x^2 + y^2}$$

$$\theta = \text{polar angle} = \tan^{-1} \frac{y}{x}$$

$$\sin \theta = \frac{y}{r} = \frac{y}{\sqrt{x^2 + y^2}}$$

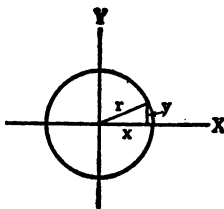
$$\cos \theta = \frac{x}{r} = \frac{x}{\sqrt{x^2 + y^2}}$$



The Circle

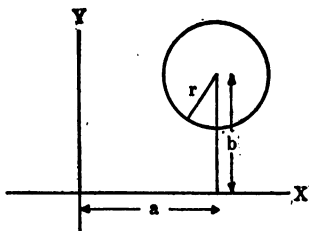
I. Circle of radius r with center at origin:

$$x^2 + y^2 = r^2$$



II. Circle of radius r with its center at the point (a, b) :

$$(x - a)^2 + (y - b)^2 = r^2$$



III. **Tangent** at the point (a, b) of the circle $x^2 + y^2 = r^2$ is

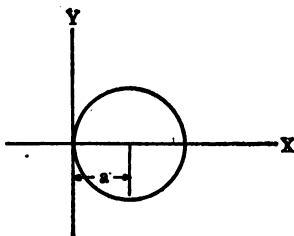
$$ax + by = r^2$$

IV. Slope equation of the **tangent** to the circle $x^2 + y^2 = r^2$ is

$$y = mx \pm r\sqrt{m^2 + 1}$$

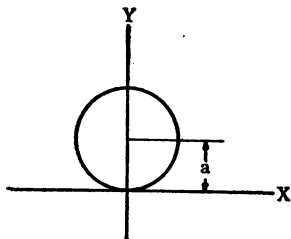
V. **Polar equation** of circle of radius a passing through the origin, and having its center on the X -axis:

$$r = 2a \cos \theta$$



VI. **Polar equation** of circle of radius a passing through the origin, and having its center on the Y -axis:

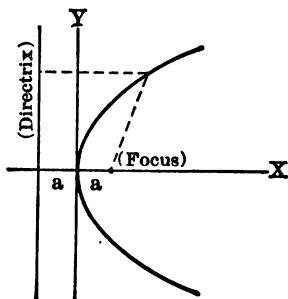
$$r = 2a \sin \theta$$



Parabola

Definition. The **parabola** is the curve generated by a point moving so as to remain always equidistant from a given fixed point and a given fixed line.

The fixed point is called the **focus**; the fixed line is called the **directrix**.



I. **Parabola** with its axis along the X -axis and vertex at origin:

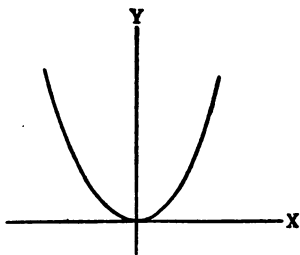
$$y^2 = 4ax$$

where a is the distance from the origin to the focus.

II. **Parabola** having its axis along the Y -axis and vertex at origin:

$$x^2 = 4ay$$

where a is the distance from the origin to the focus.



III. **General equation** of a parabola with axis parallel to the X -axis:

$$x = ay^2 + by + c$$

the **vertex** is at the point

$$\left(-\frac{b^2 - 4ac}{4a}, -\frac{b}{2a} \right)$$

IV. **General equation** of a parabola with axis parallel to the Y -axis:

$$y = ax^2 + bx + c$$

the **vertex** is at the point

$$\left(-\frac{b}{2a}, -\frac{b^2 - 4ac}{4a} \right)$$

V. **Slope equation** of the **tangent** to the parabola $y^2 = 4ax$ is

$$y = mx + \frac{a}{m}$$

VI. **Slope equation** of the **tangent** to the parabola $x^2 = 4ay$ is

$$y = mx - am^2$$

Ellipse

Definition. The **ellipse** is the curve generated by a point moving so that the sum of its distances from two fixed points is always constant. The fixed points are called the **foci**.

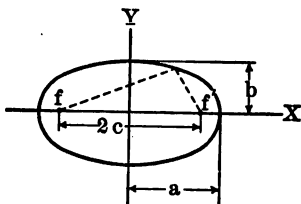
I. Equation of ellipse with **center at origin**:

$$\frac{x^2}{a^2} + \frac{y^2}{b^2} = 1$$

where a and b are one-half the major and minor axes.

II. Slope equation of the **tangent** to the ellipse $\frac{x^2}{a^2} + \frac{y^2}{b^2} = 1$ is

$$y = mx \pm \sqrt{a^2m^2 + b^2}$$



$$f, f' = \text{foci}$$

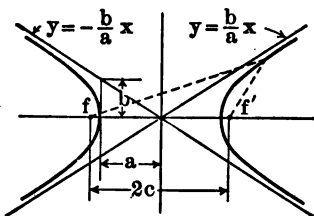
$$c = \sqrt{a^2 - b^2}$$

Hyperbola

Definition. The **hyperbola** is the curve generated by a point moving so that the difference of its distances from two fixed points is always constant.

I. Equation of hyperbola with **center at origin**:

$$\frac{x^2}{a^2} - \frac{y^2}{b^2} = 1$$

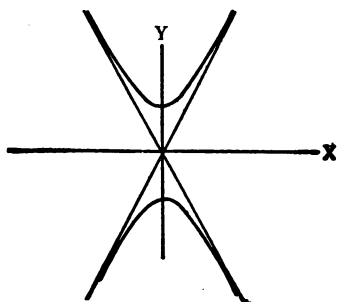


$$f, f' = \text{foci}$$

$$c = \sqrt{a^2 + b^2}$$

II. Equation of conjugate hyperbola:

$$\frac{x^2}{a^2} - \frac{y^2}{b^2} = -1$$



III. Equations of asymptotes of the hyperbola

$$\frac{x^2}{a^2} - \frac{y^2}{b^2} = 1 \text{ are}$$

$$y = \frac{b}{a}x \quad y = -\frac{b}{a}x$$

IV. Slope equation of the tangent to the hyperbola

$$\frac{x^2}{a^2} - \frac{y^2}{b^2} = 1 \text{ is}$$

$$y = mx \pm \sqrt{a^2m^2 - b^2}$$

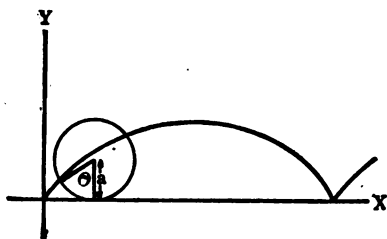
V. Slope equation of the tangent to the conjugate

$$\text{hyperbola } \frac{x^2}{a^2} - \frac{y^2}{b^2} = -1 \text{ is}$$

$$y = mx \pm \sqrt{b^2 - a^2m^2}$$

Cycloid

Definition. The cycloid is the curve generated by a point on the circumference of a circle as the circle rolls along a straight line.



$$x = a(\theta - \sin \theta)$$

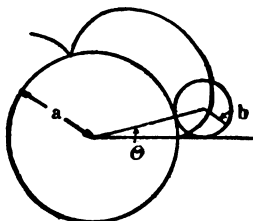
$$y = a(1 - \cos \theta)$$

$$x = a \operatorname{vers}^{-1} \frac{y}{a} - \sqrt{2ay - y^2}$$

where a is the radius of the rolling circle.

Epicycloid

Definition. The **epicycloid** is the curve generated by a fixed point on the circumference of a circle which rolls **externally** on the circumference of a fixed circle.



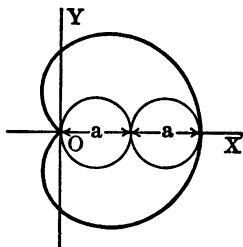
$$x = (a + b) \cos \theta - b \cos \left(\frac{a + b}{b} \theta \right)$$

$$y = (a + b) \sin \theta - b \sin \left(\frac{a + b}{b} \theta \right)$$

where a is the radius of the fixed circle, and b the radius of the rolling circle.

Cardioid

The **cardioid** is an epicycloid, with the radius of the fixed circle equal to that of the rolling circle.



$$r = a(1 + \cos \theta)$$

$$x = a \cos \theta (1 + \cos \theta)$$

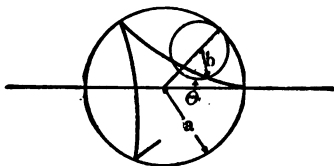
$$y = a \sin \theta (1 + \cos \theta)$$

$$\text{Area} = \frac{3\pi a^2}{2}$$

$$\text{Length} = 8a$$

Hypocycloid

Definition. The **hypocycloid** is the curve generated by a point on a circle which rolls **internally** along the circumference of a fixed circle.



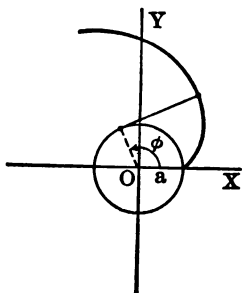
$$x = (a - b) \cos \theta + b \cos \left(\frac{a - b}{b} \theta \right)$$

$$y = (a - b) \sin \theta - b \sin \left(\frac{a - b}{b} \theta \right)$$

where a is the radius of the fixed circle and b the radius of the rolling circle.

Involute

The **involute** of a circle is the curve traced by the end of a taut string which is unwound from the circumference of a fixed circle.

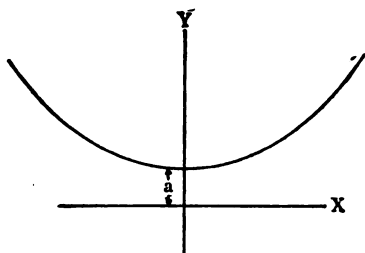


$$x = a (\cos \phi + \phi \sin \phi)$$

$$y = a (\sin \phi - \phi \cos \phi)$$

The Catenary

The **catenary** is the curve which a heavy cord or perfectly flexible chain of uniform density forms, due



to its own weight, when freely suspended between two points.

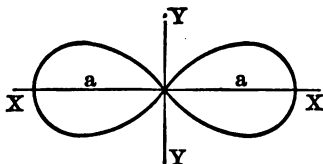
$$y = \frac{a}{2} \left(e^{\frac{x}{a}} + e^{-\frac{x}{a}} \right) = a \cosh \frac{x}{a}$$

Lemniscate

$$r^2 = a^2 \cos 2\theta$$

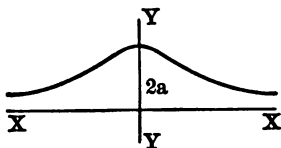
$$(x^2 + y^2)^2 = a^2(x^2 - y^2)$$

$$\text{Area} = a^2$$



Witch of Agnes

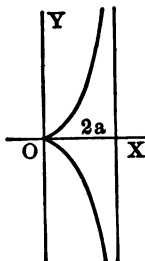
$$y = \frac{8a^3}{x^2 + 4a^2}$$



Cissoid

$$y^2 = \frac{x^3}{2a - x}$$

$$r = 2a \tan \theta \sin \theta$$

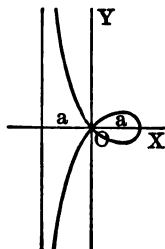


Srophoid

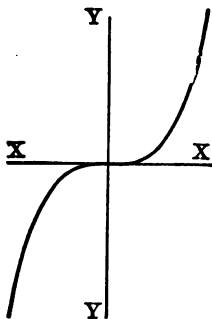
$$y^2 = x^2 \left(\frac{a-x}{a+x} \right)$$

$$r = a (\cos \theta - \sin \theta \tan \theta)$$

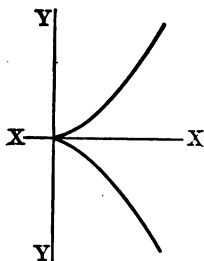
$$\text{Area of loop} = \frac{a^2}{2} (4 - \pi)$$

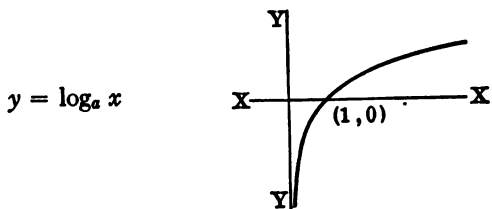
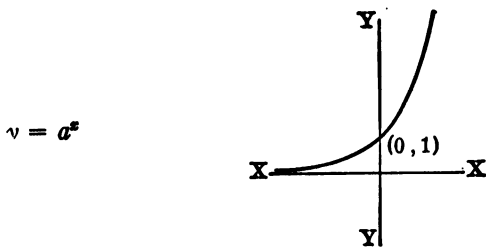
**Cubical Parabola**

$$a^2 y = x^3$$

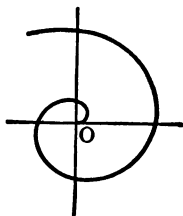
**Semi-cubical Parabola**

$$a y^2 = x^3$$



Logarithmic Curve**Exponential Curve****Spiral of Archimedes**

$$r = a\theta$$

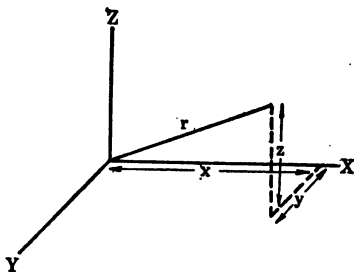


SOLID ANALYTIC GEOMETRY

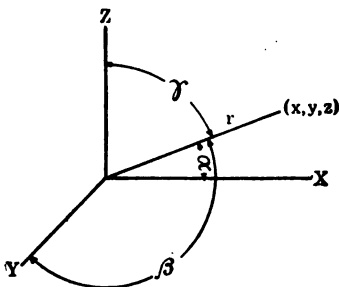
The **direction cosines** of a line in space passing through the origin are the cosines of the angles which the line makes with the rectangular coördinate axes. The direction cosines of **any line** in space are the direction cosines of a line parallel to it and passing through the origin.

I. **Distance from the point (x, y, z) to the origin:**

$$r = \sqrt{x^2 + y^2 + z^2}$$



II. The **direction cosines** of the line from the point (x, y, z) to the origin are:



$$\cos \alpha = \frac{x}{r} = \frac{x}{\sqrt{x^2 + y^2 + z^2}}$$

$$\cos \beta = \frac{y}{r} = \frac{y}{\sqrt{x^2 + y^2 + z^2}}$$

$$\cos \gamma = \frac{z}{r} = \frac{z}{\sqrt{x^2 + y^2 + z^2}}$$

III. The **sum of the squares** of the direction cosines of a line is equal to 1,

$$\cos^2 \alpha + \cos^2 \beta + \cos^2 \gamma = 1$$

IV. **Distance between the points** (x, y, z) and (x', y', z') :

$$d = \sqrt{(x - x')^2 + (y - y')^2 + (z - z')^2}$$

V. **Direction cosines of a line joining the points** (x, y, z) and (x', y', z') :

$$\cos \alpha = \frac{x - x'}{d} = \frac{x - x'}{\sqrt{(x - x')^2 + (y - y')^2 + (z - z')^2}}$$

$$\cos \beta = \frac{y - y'}{d} = \frac{y - y'}{\sqrt{(x - x')^2 + (y - y')^2 + (z - z')^2}}$$

$$\cos \gamma = \frac{z - z'}{d} = \frac{z - z'}{\sqrt{(x - x')^2 + (y - y')^2 + (z - z')^2}}$$

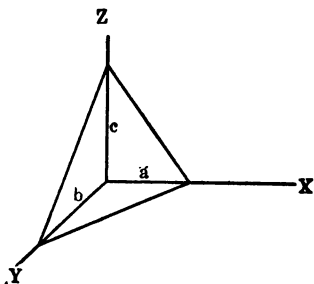
VI. The **angle between two lines** in terms of their direction cosines:

$$\cos \theta = \cos \alpha \cos \alpha' + \cos \beta \cos \beta' + \cos \gamma \cos \gamma'$$

VII. **Intercept equation** of a plane:

$$\frac{x}{a} + \frac{y}{b} + \frac{z}{c} = 1$$

where a , b , and c are the intercepts of the plane on the X , Y , and Z axes.



VIII. **General equation of a plane:**

$$Ax + By + Cz + D = 0$$

IX. **Distance from the point** (x', y', z') **to the plane** $Ax + By + Cz + D = 0$:

$$d = \frac{Ax' + By' + Cz' + D}{\pm \sqrt{A^2 + B^2 + C^2}}$$

X. **Straight line through the two points** (x'', y'', z'') **and** (x', y', z') :

$$\frac{x - x'}{x'' - x'} = \frac{y - y'}{y'' - y'} = \frac{z - z'}{z'' - z'}$$

XI. **Straight line through the point** (x', y', z') , **and making the angles** α , β , **and** γ **with the coordinate axes:**

$$\frac{x - x'}{\cos \alpha} = \frac{y - y'}{\cos \beta} = \frac{z - z'}{\cos \gamma}$$

XII. **General equation of a straight line is given by the equations of two intersecting planes:**

$$\begin{aligned} A'x + B'y + C'z + D' &= 0 \\ A''x + B''y + C''z + D'' &= 0 \end{aligned}$$

CALCULUS

Application of Differential Calculus

The following list includes some of the principal formulæ necessary for the solution of geometrical and physical problems, relating to any curve $y = f(x)$.

Rectangular Coördinates:

Slope of the tangent at the point $(x, y) = \frac{dy}{dx}$

Slope of the normal = $-\frac{dx}{dy}$

Equation of the tangent at the point (x_0, y_0) , x_0 and y_0 being the coördinates of the given point, is

$$y_0 - y = \frac{dy_0}{dx_0}(x_0 - x)$$

Equation of the normal at (x_0, y_0) is

$$(y_0 - y) = -\frac{dx_0}{dy_0}(x_0 - x)$$

The intercept of the tangent on the X -axis is $x - y \frac{dx}{dy}$

The intercept of the tangent on the Y -axis is $y - x \frac{dy}{dx}$

The intercept of the normal on the X -axis is $x + y \frac{dy}{dx}$

The intercept of the normal on the Y -axis is $y + x \frac{dx}{dy}$

Length of the tangent from its point of contact with the curve to the X -axis is

$$y \sqrt{1 + \left(\frac{dx}{dy}\right)^2}$$

Length of the tangent from its point of contact with the curve to the Y -axis is

$$x \sqrt{1 + \left(\frac{dy}{dx}\right)^2}$$

Length of the normal from its point of contact with the curve to the X -axis is

$$y \sqrt{1 + \left(\frac{dy}{dx}\right)^2}$$

Length of the normal from its point of contact with the curve to the Y -axis is

$$x \sqrt{1 + \left(\frac{dx}{dy}\right)^2}$$

Length of the subtangent = $y \frac{dx}{dy}$

Length of the subnormal = $y \frac{dy}{dx}$

Differential length of the arc = $ds = \sqrt{(dx)^2 + (dy)^2}$

$$= dy \sqrt{1 + \left(\frac{dx}{dy}\right)^2} = dx \sqrt{1 + \left(\frac{dy}{dx}\right)^2}$$

Radius of curvature = $\frac{\left[1 + \left(\frac{dy}{dx}\right)^2\right]^{\frac{3}{2}}}{\frac{d^2y}{dx^2}}$

Curvature is the reciprocal of radius of curvature.

Length of the perpendicular from the origin on the tangent (to the curve) is

$$\frac{x \frac{dy}{dx} - y}{\sqrt{1 + \left(\frac{dy}{dx}\right)^2}}$$

Polar Coördinates:

$\tan \psi = r \frac{d\theta}{dr}$, where ψ is the angle between the radius vector and that part of the tangent to the curve at (r, θ) drawn back toward the initial line.

$$\text{Length of polar subtangent} = r^2 \frac{d\theta}{dr}$$

$$\text{Length of polar subnormal} = \frac{dr}{d\theta}$$

$$\begin{aligned} \text{Differential length of arc} &= ds = \sqrt{(dr)^2 + r^2 (d\theta)^2} \\ &= dr \sqrt{1 + r^2 \left(\frac{d\theta}{dr}\right)^2} = d\theta \sqrt{r^2 + \left(\frac{dr}{d\theta}\right)^2} \end{aligned}$$

Length of the perpendicular from the pole on the tangent = $p = r^2 \frac{d\theta}{ds}$, also,

$$\frac{1}{p^2} = \frac{1}{r^2} + \frac{1}{r^4} \left(\frac{dr}{d\theta}\right)^2$$

Formulæ of Differential Calculus

$$d(au) = a du$$

$$d(u + v) = du + dv$$

$$d(uv) = v du + u dv$$

$$d\left(\frac{u}{v}\right) = \frac{v du - u dv}{v^2}$$

$$d(x^n) = nx^{n-1} dx$$

$$d(x^y) = yx^{y-1} dx + x^y \log_e x dy$$

$$d(e^x) = e^x dx$$

$$d(a^u) = a^u \log_e a du$$

$$d(\log_e x) = \frac{1}{x} dx$$

$$d(\sin x) = \cos x \, dx$$

$$d(\cos x) = -\sin x \, dx$$

$$d(\tan x) = \sec^2 x \, dx$$

$$d(\cot x) = -\operatorname{cosec}^2 x \, dx$$

$$d(\sec x) = \sec x \tan x \, dx$$

$$d(\operatorname{cosec} x) = -\operatorname{cosec} x \cot x \, dx$$

$$d(\sin^{-1} x) = \frac{dx}{\sqrt{1-x^2}}$$

$$d(\cos^{-1} x) = -\frac{dx}{\sqrt{1-x^2}}$$

$$d(\tan^{-1} x) = \frac{dx}{1+x^2}$$

$$d(\cot^{-1} x) = -\frac{dx}{1+x^2}$$

$$d(\sec^{-1} x) = \frac{dx}{x\sqrt{x^2-1}}$$

$$d(\operatorname{cosec}^{-1} x) = -\frac{dx}{x\sqrt{x^2-1}}$$

Maxima and Minima

The **maximum** or **minimum** values of a given function $y = f(x)$ are obtained as follows:

- (1) Find the first derivative $\frac{dy}{dx}$ and equate it to zero.
- (2) Solve the resulting equation for values of x .
- (3) In order to determine whether these values of x make y maximum or minimum, obtain the second derivative $\frac{d^2y}{dx^2}$ of the given function.
- (4) Substitute separately in the expression for $\frac{d^2y}{dx^2}$

each of the values of x found above. Values of x that make $\frac{d^2y}{dx^2}$ positive correspond to minimum values of the function, and values of x that make $\frac{d^2y}{dx^2}$ negative correspond to maximum values of the function.

(5) Substituting these values of x in the given function $y = f(x)$, we obtain the maximum or minimum values of y .

Illustrative Example. Find the values of x which will make the function $y = 6x + 3x^2 - 4x^3$ a maximum or a minimum, and find the corresponding values of the function y .

(1) The first derivative of y is

$$\frac{dy}{dx} = 6 + 6x - 12x^2$$

(2) The values of x which make y maximum or minimum will make $\frac{dy}{dx} = 0$; therefore

$$6 + 6x - 12x^2 = 0, \quad \text{or} \quad x^2 - \frac{1}{2}x = \frac{1}{2}$$

solving, $x = \frac{1}{4} \pm \frac{3}{4} = +1$ or $-\frac{1}{2}$

Hence, the maximum or minimum values of y must occur when $x = 1$ or $-\frac{1}{2}$.

(3) To determine whether these values are maxima or minima, we obtain the second derivative of y ; thus:

$$\frac{d^2y}{dx^2} = 6 - 24x$$

(4) When $x = 1$, $\frac{d^2y}{dx^2} = -18$, which corresponds to a maximum value of y .

When $x = -\frac{1}{2}$, $\frac{d^2y}{dx^2} = +18$, which corresponds to a minimum value of y .

(5) Substituting these values of x in the given function, we have

when $x = 1$, $y = 6 + 3 - 4 = 5$, a maximum

when $x = -\frac{1}{2}$, $y = -3 + \frac{3}{4} + \frac{1}{2} = -\frac{7}{4}$, a minimum

Taylor's and Maclaurin's Series

Taylor's Series:

$$f(x) = f(a) + \frac{(x-a)}{1!} f'(a) + \frac{(x-a)^2}{2!} f''(a) + \frac{(x-a)^3}{3!} f'''(a) + \dots$$

or

$$f(a+x) = f(a) + \frac{x}{1!} f'(a) + \frac{x^2}{2!} f''(a) + \frac{x^3}{3!} f'''(a) + \dots$$

where $f(a)$ denotes the value of the function when a is substituted for x , $f'(a)$ the value of the first derivative when a is substituted for x , $f''(a)$ the value of the second derivative when a is substituted for x , etc.

Illustrative Examples. Expand $\cos(a+x)$ in powers of x . Here

$$f(a+x) = \cos(a+x)$$

$$\text{Placing } x = 0, \quad f(a) = \cos a$$

$$f'(a+x) = -\sin(a+x), \quad f'(a) = -\sin a$$

$$f''(a+x) = -\cos(a+x), \quad f''(a) = -\cos a$$

$$f'''(a+x) = \sin(a+x), \quad f'''(a) = \sin a$$

Substituting in Taylor's formula,

$$\cos(a+x) = \cos a - \frac{x}{1!} \sin a - \frac{x^2}{2!} \cos a + \frac{x^3}{3!} \sin a + \dots$$

Maclaurin's Series.

$$f(x) = f(0) + \frac{x}{1!} f'(0) + \frac{x^2}{2!} f''(0) + \frac{x^3}{3!} f'''(0) + \dots$$

where $f(0)$ denotes the value of the function when 0 is substituted for x , $f'(0)$ the value of the first derivative when 0 is substituted for x , etc.

Illustrative Example. Expand $\cos x$ in powers of x .

Here $f(x) = \cos x$

$$f(0) = \cos 0 = 1$$

$$f'(x) = -\sin x, \quad f'(0) = 0$$

$$f''(x) = -\cos x, \quad f''(0) = -1$$

$$f'''(x) = \sin x, \quad f'''(0) = 0$$

$$f^{iv}(x) = \cos x, \quad f^{iv}(0) = 1$$

Substituting in Maclaurin's formula,

$$\cos x = 1 - \frac{x^2}{2!} + \frac{x^4}{4!} - \dots$$

APPLICATION OF INTEGRAL CALCULUS

Lengths of Curves

Rectangular Coördinates:

$$\text{length of curve} = s = \int_a^b \sqrt{1 + \left(\frac{dy}{dx}\right)^2} dx$$

From the equation of the given curve, find y in terms

of x ; then differentiate in order to obtain $\frac{dy}{dx}$, and substitute its value in the formula. The lower limit a is the initial value of x , and the upper limit b the final value of x .

Or, similarly, by solving for x in terms of y , and obtaining $\frac{dx}{dy}$, the length of the curve is given by the formula

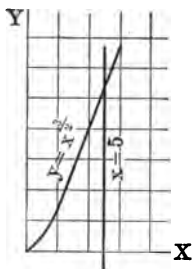
$$s = \int_c^d \sqrt{1 + \left(\frac{dx}{dy}\right)^2} dy$$

where c and d are the initial and final values of y .

Illustrative Example. Find the length of the arc of the semi-cubical parabola $y^2 = x^3$ from the origin to the ordinate $x = 5$.

$$y = x^{3/2}, \quad \frac{dy}{dx} = \frac{3}{2}x^{1/2},$$

$$\left(\frac{dy}{dx}\right)^2 = \frac{9x}{4}$$



The required length of arc is

$$\begin{aligned} S &= \int_0^5 \sqrt{1 + \left(\frac{dy}{dx}\right)^2} dx = \int_0^5 \sqrt{1 + \frac{9x}{4}} dx \\ &= \frac{8}{27} \sqrt{\left(1 + \frac{9x}{4}\right)^3} \Big|_0^5 = \frac{335}{27} \end{aligned}$$

Polar Coördinates:

$$\text{length of curve} = s = \int_a^b \sqrt{1 + r^2 \left(\frac{d\theta}{dr}\right)^2} dr$$

where a and b are the limiting values of r .

Or,

$$\text{length of curve} = s = \int_{\theta'}^{\theta''} \sqrt{r^2 + \left(\frac{dr}{d\theta}\right)^2} d\theta$$

where θ' and θ'' are the limiting values of θ .

Plane Areas

Rectangular Coördinates:

The area included between a curve, the X -axis, and the vertical lines $x = a$ and $x = b$ is

$$\text{area} = A = \int_a^b y dx$$

The value of y in terms of x is found from the given equation and substituted in the formula. The initial value of x is a , and the final value b .

Similarly, the area included between a curve, the Y -axis, and the horizontal lines $y = c$ and $y = d$ is

$$\text{area} = A = \int_c^d x dy$$

where c and d are the limits of y .

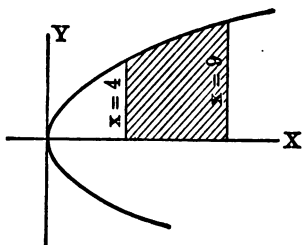
Illustrative Example.

Find the area bounded by the parabola $y^2 = 4x$, the axis of X , and the lines $x = 4$, $x = 9$.

$$y^2 = 4x, \quad y = 2\sqrt{x}$$

The required area is

$$\begin{aligned} A &= \int_4^9 y dx = \int_4^9 2\sqrt{x} dx \\ &= \left. \frac{4}{3} x^{3/2} \right]_4^9 = 25\frac{1}{3} \end{aligned}$$



Polar Coördinates:

The area included between a given curve and two given radii is

$$\text{area} = A = \frac{1}{2} \int_{\theta'}^{\theta''} r^2 d\theta$$

where θ'' and θ' are the limiting values of θ .

Areas of Surfaces of Revolution

For revolution about the X -axis,

$$\text{area} = A = 2\pi \int_a^b y \sqrt{1 + \left(\frac{dy}{dx}\right)^2} dx$$

where the value of $\left(\frac{dy}{dx}\right)$ is found from the given equation. The initial value of x is a , and the final value b .

For revolution about the Y -axis,

$$\text{area} = A = 2\pi \int_c^d x \sqrt{1 + \left(\frac{dx}{dy}\right)^2} dy$$

where c and d are the limiting values of y .

Volumes of Solids of Revolution

Rectangular Coördinates:

$$\text{volume} = V_x = \pi \int_a^b y^2 dx$$

is the formula for the volume generated by revolving the given curve about the X -axis. The limiting values of x are a and b .

Similarly, the volume generated by revolving the plane figure about the Y -axis equals

$$V_y = \pi \int_c^d x^2 dy$$

where c and d are the initial and final values of y .

Polar Coördinates:

When the plane figure is revolved about the X -axis, the volume generated is

$$V_x = 2\pi \int \int r^2 \sin \theta \, d\theta \, dr$$

For revolution about the Y -axis, the volume generated is

$$V_y = 2\pi \int \int r^2 \cos \theta \, d\theta \, dr$$

INDETERMINATE FORMS

If the fraction $\frac{f(x)}{F(x)}$ gives rise to the indeterminate form $\frac{0}{0}$ or $\frac{\infty}{\infty}$, when x approaches a as a limit, the indeterminate form may be replaced by a new fraction, $\frac{f'(x)}{F'(x)}$, the numerator of which is equal to the derivative of the given numerator, and the new denominator is equal to the derivative of the given denominator. The value of this new fraction, as x approaches a , is the limiting value of the given fraction. If this again becomes indeterminate, it may be necessary to repeat the process several times.

Example. Find the limiting value, when $x = 1$, of the fraction

$$\frac{x^2 + x - 2}{x^2 - 1}$$

$$\frac{f(x)}{F(x)} = \frac{x^2 + x - 2}{x^2 - 1} = \frac{0}{0}, \text{ when } x = 1$$

$$\frac{f'(x)}{F'(x)} = \frac{2x + 1}{2x} = \frac{3}{2}, \text{ when } x = 1$$

Hence, the required limiting value is $\frac{3}{2}$.

SOLUTION OF EQUATIONS

Algebraic equations may be solved by Newton's method of approximation. Thus, let it be required to solve an equation of the form $Ax^3 + Bx^2 + Cx = D$. Find, by trial, a number, r , nearly equal to the root sought, and let $r + h$ denote the exact value of the root, where h is a small quantity the value of which must be determined. Substituting $r + h$ for x in the given equation and neglecting all powers of h higher than the first, we have, approximately,

$$h = \frac{Ar^3 + Br^2 + Cr - D}{-3Ar^2 - 2Br - C}$$

It will be observed that the numerator of the above fraction is the first member of the given equation after D has been transposed and x changed to r , and the denominator is the **first derivative** of the numerator with its sign reversed. The correction h added, with its proper sign, to the assumed root r , gives a closer approximation to the value of x . Repeat the operation with the corrected value of r , and a second correction will be obtained which will give a nearer value of the root; two corrections generally give sufficient accuracy.

Illustration. Find a root of the equation

$$x^3 + 2x^2 + 3x = 50$$

The value of h is

$$h = \frac{r^3 + 2r^2 + 3r - 50}{-3r^2 - 4r - 3}$$

By trial, we find that x is nearly equal to 3. On substituting 3 for r , we have

$$h = -\frac{2}{21} = -0.1, \text{ approximately}$$

Hence, $x = 2.9$, nearly. If we substitute this new value of r , the new value of h equals $+0.00228$. Hence $x = 2.90228$. If we repeat the operation with this last value of r , the value of h is then found to be $+0.0000034$. Hence $x = 2.9022834$.

CURVE TRACING

The usual method of tracing curves consists in assigning a series of different values to one of the variables, and calculating the corresponding series of values of the other, thus determining a definite number of points on the curve. By drawing a curve through these points, we obtain a graphical representation of the given equation.

The **general form** and **peculiarities** of the curve can be easily determined and sketched by the following steps:

(1) If possible, solve the equation of the given curve for one of its variables, y for example. If the equation then contains only even powers of x , it is symmetrical with the Y -axis.

Or if, when solved for x , it contains only even powers of y , it is symmetrical with the X -axis.

(2) Find the points in which the curve cuts the axes by solving the equation of the given curve in turn with the equations $x = 0$ and $y = 0$.

(3) Find the values of x , if any, which make y infinite; similarly, test for infinite values of x .

(4) Find the value of the first derivative $\frac{dy}{dx}$; and

thence deduce the maximum and minimum points of the curve.

In tracing polar curves, write the equation, if possible, in the form $r = f(\theta)$; and give θ such values as make r easily found, as for example, $0, \frac{1}{2}\pi, \pi, \frac{3}{2}\pi$, etc.

Putting $\frac{dr}{d\theta} = 0$, we find the values of θ for which r is a maximum or minimum.

METHODS OF INTEGRATION

(By parts, substitution, etc.)

When the numerator of a fraction contains a variable to an **equal** or a **higher** power than the denominator, the fraction must be reduced to a mixed quantity (by actually dividing the denominator into the numerator) before it can be integrated.

If an expression cannot be integrated by the formulæ given in the table of integrals, one of the following methods may be used to obtain a solution.

Partial Fractions

A fraction may be resolved into partial fractions, which can be integrated separately.

Example. To integrate

$$\frac{1}{(x+a)(x+b)} dx$$

Let

$$\frac{1}{(x+a)(x+b)} = \frac{A}{x+a} + \frac{B}{x+b}$$

where we must determine A and B .

Clearing of fractions,

$$1 = A(x+b) + B(x+a) = (A+B)x + (bA + aB)$$

The coefficients of like powers of x on both sides of the equation are equal; therefore,

$$A + B = 0$$

$$bA + aB = 1$$

whence $A = \frac{1}{b-a}$ and $B = \frac{1}{a-b}$

and

$$\int \frac{1}{(x+a)(x+b)} dx = \int \left(\frac{1}{b-a} \right) \frac{1}{(x+a)} dx + \int \left(\frac{1}{a-b} \right) \frac{1}{(x+b)} dx$$

These forms are now integrable by the table of integrals, the result being

$$\int \frac{1}{(x+a)(x+b)} dx = \frac{1}{b-a} \log(x+a) + \frac{1}{a-b} \log(x+b) + C$$

where C is the constant of integration.

Integration by Parts

To integrate by parts, apply the formula

$$\int u dv = uv - \int v du$$

The method of integration by parts is most effective in dealing with the integration of **products**, involving logarithms, and trigonometric and inverse circular functions.

Generally, the most complicated quantity which can be integrated directly by one of the fundamental formulæ (see Table of Integrals, page 57) is equated, with the differential, to dv , and the remaining part is equated to u .

Example. To find

$$\int x \log(x) dx$$

Let $u = \log x$ and $dv = x dx$

then $du = \frac{dx}{x}$ $v = \int x dx = \frac{x^2}{2}$

Substituting in the formula

$$\int u dv = uv - \int v du$$

we have

$$\begin{aligned} \int x \log(x) dx &= \log(x) \cdot \frac{x^2}{2} - \int \frac{x^2}{2} \frac{dx}{x} \\ &= \frac{x^2}{2} \log(x) - \frac{x^2}{4} + C \end{aligned}$$

Integration by Substitution

I. Differentials containing fractional powers of x may be integrated by the substitution

$$x = z^n$$

where n is the least common denominator of the fractional exponents of x .

II. Expressions involving only fractional powers of $(a + bx)$ may be rationalized by the substitution

$$(a + bx) = z^n$$

where n is the least common denominator of the fractional exponents of $(a + bx)$.

III. To integrate expressions containing

$$\sqrt{x^2 + ax + b},$$

use the substitution

$$\sqrt{x^2 + ax + b} = z - x$$

IV. Expressions containing $\sqrt{-x^2 + ax + b}$ may be rationalized by the substitution

$$\sqrt{-x^2 + ax + b} = (x - \theta) z$$

where $(x - \theta)$ is a factor of $(-x^2 + ax + b)$.

V. A differential containing $\sin x$ and $\cos x$ can be transformed by means of the substitution

$$\tan \frac{x}{2} = z$$

from which

$$\sin x = \frac{2z}{1+z^2} \quad \cos x = \frac{1-z^2}{1+z^2} \quad dx = \frac{2dz}{1+z^2}$$

VI. A very useful substitution is

$$x = \frac{1}{z}$$

VII. Differentials involving $\sqrt{a^2 - x^2}$ may be rationalized by the substitution

$$x = a \sin \theta$$

VIII. Differentials involving $\sqrt{a^2 + x^2}$ may be rationalized by the substitution

$$x = a \tan \theta$$

IX. Differentials involving $\sqrt{x^2 - a^2}$ may be rationalized by the substitution

$$x = a \sec \theta$$

Reduction Formulæ

The purpose of the following reduction formulæ is to simplify an integral of the form

$$\int x^m (a + bx^n)^p dx$$

$$\int x^m (a + bx^n)^p dx = \frac{x^{m-n+1} (a + bx^n)^{p+1}}{(np + m + 1)b}$$

$$- \frac{(m - n + 1)a}{(np + m + 1)b} \int x^{m-n} (a + bx^n)^p dx$$

This formula enables us to lower the exponent of x by n , without affecting the exponent of $(a + bx^n)$.

Method fails when $(np + m + 1) = 0$.

$$\text{II. } \int x^m (a + bx^n)^p dx = \frac{x^{m+1} (a + bx^n)^p}{(np + m + 1)} \\ + \frac{npa}{(np + m + 1)} \int x^m (a + bx^n)^{p-1} dx$$

By this formula, the exponent of $(a + bx^n)$ is lowered by 1, without affecting the exponent of x .

Method fails when $(np + m + 1) = 0$.

$$\text{III. } \int x^m (a + bx^n)^p dx = \frac{x^{m+1} (a + bx^n)^{p+1}}{(m+1)a} \\ - \frac{(np + m + 1 + n)b}{(m+1)a} \int x^{m+n} (a + bx^n)^p dx$$

By this formula, the exponent of x is increased by n , without affecting the exponent of $(a + bx^n)$.

Method fails when $m = -1$.

$$\text{IV. } \int x^m (a + bx^n)^p dx = -\frac{x^{m+1} (a + bx^n)^{p+1}}{n(p+1)a} \\ + \frac{(np + n + m + 1)}{n(p+1)a} \int x^m (a + bx^n)^{p+1} dx$$

This formula enables us to increase the exponent of $(a + bx^n)$ by 1, without affecting the exponent of x .

Method fails when $p = -1$.

TABLE OF INTEGRALS

Fundamental Forms

$$\int x^n dx = \frac{x^{n+1}}{n+1}$$

$$\int \frac{dx}{x} = \log x$$

$$\int e^x dx = e^x$$

$$\int a^x dx = \frac{a^x}{\log_e a}$$

$$\int \frac{dx}{1+x^2} = \tan^{-1} x$$

$$\int \frac{dx}{\sqrt{1-x^2}} = \sin^{-1} x$$

$$\int \frac{dx}{x\sqrt{x^2-1}} = \sec^{-1} x$$

$$\int \sin x dx = -\cos x$$

$$\int \cos x dx = \sin x$$

$$\int \tan x dx = \log (\sec x)$$

$$\int \cot x dx = \log (\sin x)$$

$$\int \sec x dx = \log \left[\tan \left(\frac{x}{2} + \frac{\pi}{4} \right) \right]$$

$$\int \operatorname{cosec} x dx = \log \left(\tan \frac{x}{2} \right)$$

$$\int \tan x \sec x dx = \sec x$$

$$\int \cot x \operatorname{cosec} x dx = -\operatorname{cosec} x$$

$$\int \sec^2 x dx = \tan x$$

$$\int \operatorname{cosec}^2 x dx = -\cot x$$

Expressions involving $(a + bx)$:

$$\int \frac{dx}{(a + bx)} = \frac{1}{b} \log (a + bx)$$

$$\int \frac{dx}{(a+bx)^2} = -\frac{1}{b(a+bx)}$$

$$\int \frac{x dx}{(a+bx)} = \frac{1}{b^2}[a+bx - a \log(a+bx)]$$

$$\int \frac{x dx}{(a+bx)^2} = \frac{1}{b^2} \left[\log(a+bx) + \frac{a}{a+bx} \right]$$

$$\int \frac{x^2 dx}{a+bx} = \frac{1}{b^3} \left[\frac{1}{2}(a+bx)^2 - 2a(a+bx) + a^2 \log(a+bx) \right]$$

$$\int \frac{x^2 dx}{(a+bx)^2} = \frac{1}{b^3} \left[(a+bx) - 2a \log(a+bx) - \frac{a^2}{a+bx} \right]$$

$$\int \frac{dx}{(a+bx)^3} = -\frac{1}{2b(a+bx)^2}$$

$$\int \frac{x dx}{(a+bx)^3} = \frac{1}{b^2} \left[-\frac{1}{a+bx} + \frac{a}{2(a+bx)^2} \right]$$

$$\int \frac{x^2 dx}{(a+bx)^3} = \frac{1}{b^3} \left[\log(a+bx) + \frac{2a}{a+bx} - \frac{a^2}{2(a+bx)^2} \right]$$

$$\int \frac{dx}{x(a+bx)} = -\frac{1}{a} \log \frac{a+bx}{x}$$

$$\int \frac{dx}{x(a+bx)^2} = \frac{1}{a(a+bx)} - \frac{1}{a^2} \log \frac{a+bx}{x}$$

$$\int \frac{dx}{x^2(a+bx)} = -\frac{1}{ax} + \frac{b}{a^2} \log \frac{a+bx}{x}$$

$$\int \frac{dx}{x^2(a+bx)^2} = -\frac{a+2bx}{a^2x(a+bx)} + \frac{2b}{a^3} \log \frac{a+bx}{x}$$

$$\int (a+bx)^n dx = \frac{1}{b(n+1)} (a+bx)^{n+1}$$

$$\int x(a+bx)^n dx = \frac{1}{b^2(n+2)} (a+bx)^{n+2}$$

$$-\frac{a}{b^2(n+1)} (a+bx)^{n+1}$$

$$\int x^2 (a + bx)^n dx = \frac{1}{b^3} \left[\frac{(a+bx)^{n+3}}{n+3} - 2a \frac{(a+bx)^{n+2}}{n+2} + a^2 \frac{(a+bx)^{n+1}}{n+1} \right]$$

$$\int \frac{dx}{(a+bx)(c+dx)} = \frac{1}{ad-bc} \log \frac{c+dx}{a+bx}$$

$$\int \frac{dx}{(a+bx)^2(c+dx)} = \frac{1}{ad-bc} \left[\frac{1}{a+bx} + \frac{d}{ad-bc} \log \frac{c+dx}{a+bx} \right]$$

Expressions involving $(a+bx^2)$ or $(a^2 \pm x^2)$:

$$\int \frac{dx}{a^2+x^2} = \frac{1}{a} \tan^{-1} \frac{x}{a}$$

$$\int \frac{dx}{a^2-x^2} = \frac{1}{2a} \log \frac{a+x}{a-x}$$

$$\int \frac{dx}{a+bx^2} = \frac{1}{\sqrt{ab}} \tan^{-1} \left(x \sqrt{\frac{b}{a}} \right) \quad \text{or}$$

$$\int \frac{dx}{a+bx^2} = \frac{1}{2\sqrt{-ab}} \log \frac{\sqrt{a}+x\sqrt{-b}}{\sqrt{a}-x\sqrt{-b}} \quad \text{if } a > 0, b < 0$$

$$\int \frac{dx}{(a+bx^2)^2} = \frac{x}{2a(a+bx^2)} + \frac{1}{2a} \int \frac{dx}{a+bx^2}$$

$$\int \frac{x dx}{a+bx^2} = \frac{1}{2b} \log \left(x^2 + \frac{a}{b} \right)$$

$$\int \frac{x^2 dx}{a+bx^2} = \frac{x}{b} - \frac{a}{b} \int \frac{dx}{a+bx^2}$$

$$\int \frac{dx}{x(a+bx^2)} = \frac{1}{2a} \log \frac{x^2}{a+bx^2}$$

$$\int \frac{dx}{(a+bx^2)^n} = \frac{1}{2(n-1)a} \frac{x}{(a+bx^2)^{n-1}} + \frac{2n-3}{2(n-1)a} \int \frac{dx}{(a+bx^2)^{n-1}} \quad (n \text{ integer } > 1)$$

$$\int (a + bx^2)^n x dx = \frac{1}{2b} \frac{(a + bx^2)^{n+1}}{n+1}$$

$$\int \frac{x^2 dx}{(a + bx^2)^n} = -\frac{1}{2(n-1)b} \frac{x}{(a + bx^2)^{n-1}} + \frac{1}{2(n-1)b} \int \frac{dx}{(a + bx^2)^{n-1}} \quad (n \text{ integer } > 1)$$

$$\int \frac{dx}{x^2 (a + bx^2)^n} = \frac{1}{a} \int \frac{dx}{x^2 (a + bx^2)^{n-1}} - \frac{b}{a} \int \frac{dx}{(a + bx^2)^n} \quad (n \text{ positive integer})$$

Expressions involving $\sqrt{a + bx}$:

$$\int \sqrt{a + bx} dx = \frac{2}{3b} \sqrt{(a + bx)^3}$$

$$\int x \sqrt{a + bx} dx = -\frac{2(2a - 3bx) \sqrt{(a + bx)^3}}{15b^2}$$

$$\int x^2 \sqrt{a + bx} dx = \frac{2(8a^2 - 12abx + 15b^2x^2) \sqrt{(a + bx)^3}}{105b^3}$$

$$\int \frac{\sqrt{a + bx}}{x} dx = 2\sqrt{a + bx} + a \int \frac{dx}{x \sqrt{a + bx}}$$

$$\int \frac{dx}{\sqrt{a + bx}} = \frac{2\sqrt{a + bx}}{b}$$

$$\int \frac{x dx}{\sqrt{a + bx}} = -\frac{2(2a - bx)}{3b^2} \sqrt{a + bx}$$

$$\int \frac{x^2 dx}{\sqrt{a + bx}} = \frac{2(8a^2 - 4abx + 3b^2x^2)}{15b^3} \sqrt{a + bx}$$

$$\int \frac{dx}{x \sqrt{a + bx}} = \frac{1}{\sqrt{a}} \log \left[\frac{\sqrt{a + bx} - \sqrt{a}}{\sqrt{a + bx} + \sqrt{a}} \right] \quad (a \text{ pos.})$$

or

$$\int \frac{dx}{x \sqrt{a + bx}} = \frac{2}{\sqrt{-a}} \tan^{-1} \sqrt{\frac{a + bx}{-a}} \quad (a \text{ neg.})$$

$$\int \frac{dx}{x^2 \sqrt{a+bx}} = -\frac{\sqrt{a+bx}}{ax} - \frac{b}{2a} \int \frac{dx}{x \sqrt{a+bx}}$$

$$\int \frac{dx}{(a+bx)(c+dx)} = \frac{1}{ad-bc} \log \frac{c+dx}{a+bx}$$

$$\int \frac{c+dx}{\sqrt{a+bx}} dx = \frac{2}{3b^2} (3bc - 2ad + bdx) \sqrt{a+bx}$$

$$\int \frac{\sqrt{a+bx}}{c+dx} dx = \frac{2\sqrt{a+bx}}{d} - \frac{2}{d} \sqrt{\frac{bc-ad}{d}} \tan^{-1} \sqrt{\frac{d(a+bx)}{bc-ad}} \quad (d \text{ pos } bc > ad)$$

$$\int \frac{\sqrt{a+bx}}{c+dx} dx = \frac{2\sqrt{a+bx}}{d} + \frac{1}{d} \sqrt{\frac{ad-bc}{d}} \log \left(\frac{\sqrt{d(a+bx)} - \sqrt{ad-bc}}{\sqrt{d(a+bx)} + \sqrt{ad-bc}} \right) \quad (d \text{ pos } ad > bc)$$

$$\int \frac{dx}{(c+dx) \sqrt{a+bx}} = \frac{2}{\sqrt{d} \sqrt{bc-ad}} \tan^{-1} \sqrt{\frac{d(a+bx)}{bc-ad}} \quad (d \text{ pos } bc > ad)$$

$$\int \frac{dx}{(c+dx) \sqrt{a+bx}} = \frac{1}{\sqrt{d} \sqrt{ad-bc}} \log \frac{\sqrt{d(a+bx)} - \sqrt{ad-bc}}{\sqrt{d(a+bx)} + \sqrt{ad-bc}} \quad (d \text{ pos } ad > bc)$$

Expressions involving $\sqrt{a^2 - x^2}$ or $\sqrt{a^2 + x^2}$:

$$\int \sqrt{a^2 - x^2} dx = \frac{1}{2} \left[x \sqrt{a^2 - x^2} + a^2 \sin^{-1} \frac{x}{a} \right]$$

$$\int \frac{dx}{\sqrt{a^2 - x^2}} = \sin^{-1} \frac{x}{a}$$

$$\int \frac{dx}{x \sqrt{a^2 \pm x^2}} = -\frac{1}{a} \left[\log \frac{a + \sqrt{a^2 \pm x^2}}{x} \right]$$

$$\int \frac{\sqrt{a^2 \pm x^2}}{x} dx = \sqrt{a^2 \pm x^2} - a \log \left[\frac{a + \sqrt{a^2 \pm x^2}}{x} \right]$$

$$\int \frac{x dx}{\sqrt{a^2 \pm x^2}} = \pm \sqrt{a^2 \pm x^2}$$

$$\int x \sqrt{a^2 - x^2} dx = -\frac{1}{3} \sqrt{(a^2 - x^2)^3}$$

$$\int \sqrt{(a^2 - x^2)^3} dx = \frac{x}{8} (5a^2 - 2x^2) \sqrt{a^2 - x^2} + \frac{3}{8} a^4 \sin^{-1} \frac{x}{a}$$

$$\int x^2 \sqrt{a^2 - x^2} dx = -\frac{x}{4} \sqrt{(a^2 - x^2)^3} + \frac{a^2}{8} \left[x \sqrt{a^2 - x^2} + a^2 \sin^{-1} \frac{x}{a} \right]$$

$$\int \frac{x^2 dx}{\sqrt{a^2 - x^2}} = -\frac{x}{2} \sqrt{a^2 - x^2} + \frac{a^2}{2} \sin^{-1} \frac{x}{a}$$

$$\int \frac{x^2 dx}{\sqrt{(a^2 - x^2)^3}} = \frac{x}{\sqrt{a^2 - x^2}} - \sin^{-1} \left(\frac{x}{a} \right)$$

$$\int \frac{dx}{x^2 \sqrt{a^2 - x^2}} = -\frac{\sqrt{a^2 - x^2}}{a^2 x}$$

$$\int \frac{\sqrt{a^2 - x^2}}{x^2} dx = -\frac{\sqrt{a^2 - x^2}}{x} - \sin^{-1} \frac{x}{a}$$

Expressions involving $\sqrt{x^2 + a^2}$ or $\sqrt{x^2 - a^2}$:

$$\int \sqrt{x^2 \pm a^2} dx =$$

$$\frac{1}{2} [x \sqrt{x^2 \pm a^2} \pm a^2 \log (x + \sqrt{x^2 \pm a^2})]$$

$$\int \frac{dx}{\sqrt{x^2 \pm a^2}} = \log [x + \sqrt{x^2 \pm a^2}]$$

$$\int \frac{dx}{x\sqrt{x^2 - a^2}} = \frac{1}{a} \cos^{-1} \frac{a}{x}$$

$$\int \frac{\sqrt{x^2 - a^2}}{x} dx = \sqrt{x^2 - a^2} - a \cos^{-1} \frac{a}{x}$$

$$\int \frac{x dx}{\sqrt{x^2 - a^2}} = \sqrt{x^2 - a^2}$$

$$\int \frac{x dx}{\sqrt{(x^2 \pm a^2)^3}} = -\frac{1}{\sqrt{x^2 \pm a^2}}$$

$$\int x \sqrt{x^2 \pm a^2} dx = \frac{1}{3} \sqrt{(x^2 \pm a^2)^3}$$

$$\int \sqrt{(x^2 \pm a^2)^3} dx = \frac{x}{8} (2x^2 \pm 5a^2) \sqrt{x^2 \pm a^2}$$

$$+ \frac{3a^4}{8} \log (x + \sqrt{x^2 \pm a^2})$$

$$\int \frac{dx}{\sqrt{(x^2 \pm a^2)^3}} = \frac{\pm x}{a^2 \sqrt{x^2 \pm a^2}}$$

$$\int x^2 \sqrt{x^2 \pm a^2} dx = \frac{x}{8} (2x^2 \pm a^2) \sqrt{x^2 \pm a^2}$$

$$- \frac{a^4}{8} \log (x + \sqrt{x^2 \pm a^2})$$

$$\int \frac{x^2 dx}{\sqrt{x^2 \pm a^2}} = \frac{x}{2} \sqrt{x^2 \pm a^2} \mp \frac{a^2}{2} \log (x + \sqrt{x^2 \pm a^2})$$

$$\int \frac{x^2 dx}{\sqrt{(x^2 \pm a^2)^3}} = -\frac{x}{\sqrt{x^2 \pm a^2}} + \log (x + \sqrt{x^2 \pm a^2})$$

$$\int \frac{dx}{x^2 \sqrt{x^2 \pm a^2}} = \mp \frac{\sqrt{x^2 \pm a^2}}{a^2 x}$$

$$\int \frac{\sqrt{x^2 \pm a^2} dx}{x^2} = -\frac{\sqrt{x^2 \pm a^2}}{x} + \log(x + \sqrt{x^2 \pm a^2})$$

Expressions involving $ax^2 + bx + c$.

$$\int \frac{dx}{ax^2 + bx + c} = \frac{1}{\sqrt{b^2 - 4ac}} \log \frac{(2ax + b) - \sqrt{b^2 - 4ac}}{(2ax + b) + \sqrt{b^2 - 4ac}}$$

if $b^2 > 4ac$

$$\int \frac{dx}{ax^2 + bx + c} = \frac{2}{\sqrt{4ac - b^2}} \tan^{-1} \frac{2ax + b}{\sqrt{4ac - b^2}}$$

if $(b^2 < 4ac)$

$$\int \frac{dx}{ax^2 + bx + c} = \frac{-2}{2ax + b}$$

if $b^2 = 4ac$

$$\int \frac{x dx}{ax^2 + bx + c} = \frac{1}{2a} \log(ax^2 + bx + c) - \frac{b}{2a} \int \frac{dx}{ax^2 + bx + c}$$

$$\int \frac{x^2 dx}{ax^2 + bx + c} = \frac{x}{a} - \frac{b}{2a^2} \log(ax^2 + bx + c) + \frac{b^2 - 2ac}{2a^2} \int \frac{dx}{ax^2 + bx + c}$$

Expressions involving $\sqrt{\pm ax^2 + bx + c}$:

$$\int \frac{dx}{\sqrt{ax^2 + bx + c}} = \frac{1}{\sqrt{a}} \log(2ax + b + 2\sqrt{a}\sqrt{ax^2 + bx + c})$$

$$\int \sqrt{ax^2 + bx + c} dx = \frac{2ax + b}{4a} \sqrt{ax^2 + bx + c} - \frac{b^2 - 4ac}{8a} \int \frac{dx}{\sqrt{ax^2 + bx + c}}$$

$$\int \frac{dx}{\sqrt{-ax^2 + bx + c}} = \frac{1}{\sqrt{a}} \sin^{-1} \left(\frac{2ax - b}{\sqrt{b^2 + 4ac}} \right)$$

$$\int \sqrt{-ax^2 + bx + c} dx = \frac{2ax - b}{4a} \sqrt{-ax^2 + bx + c} + \frac{b^2 + 4ac}{8a} \int \frac{dx}{\sqrt{-ax^2 + bx + c}}$$

$$\int \frac{dx}{\sqrt{(ax^2 + bx + c)^3}} = - \frac{2(2ax + b)}{(b^2 - 4ac)\sqrt{ax^2 + bx + c}}$$

Formulae involving $\sqrt{2ax - x^2}$:

$$\int \sqrt{2ax - x^2} dx = \frac{x - a}{2} \sqrt{2ax - x^2} + \frac{a^2}{2} \sin^{-1} \frac{x - a}{a}$$

$$\int x \sqrt{2ax - x^2} dx = - \frac{3a^2 + ax - 2x^2}{6} \sqrt{2ax - x^2} + \frac{a^3}{2} \text{vers}^{-1} \frac{x}{a}$$

$$\int \frac{dx}{\sqrt{2ax - x^2}} = \text{vers}^{-1} \frac{x}{a}$$

$$\int \frac{x dx}{\sqrt{2ax - x^2}} = - \sqrt{2ax - x^2} + a \text{vers}^{-1} \frac{x}{a}$$

$$\int \frac{dx}{x \sqrt{2ax - x^2}} = - \frac{\sqrt{2ax - x^2}}{ax}$$

$$\int \frac{\sqrt{2ax - x^2}}{x} dx = \sqrt{2ax - x^2} + a \text{vers}^{-1} \frac{x}{a}$$

$$\int \frac{dx}{\sqrt{(2ax - x^2)^3}} = \frac{x - a}{a^2 \sqrt{2ax - x^2}}$$

$$\int \sqrt{\frac{a+x}{b+x}} dx = \sqrt{(a+x)(b+x)}$$

$$+ (a-b) \log [\sqrt{a+x} + \sqrt{b+x}]$$

$$\int \sqrt{\frac{a-x}{b+x}} dx = \sqrt{(a-x)(b+x)} + (a+b) \sin^{-1} \sqrt{\frac{b+x}{a+b}}$$

$$\int \frac{dx}{x(a+bx^n)} = \frac{1}{an} \log \frac{x^n}{a+bx^n}$$

$$\int \frac{dx}{x\sqrt{a+bx^n}} = \frac{1}{n\sqrt{a}} \log \frac{\sqrt{(a+bx^n)} - \sqrt{a}}{\sqrt{a+bx^n} + \sqrt{a}} \quad (a \text{ pos.})$$

$$\int \frac{dx}{x\sqrt{a+bx^n}} = \frac{2}{n\sqrt{-a}} \sec^{-1} \sqrt{\frac{-bx^n}{a}} \quad (a \text{ neg.})$$

Expressions involving trigonometric forms:

$$\int \sin^2 x dx = \frac{x}{2} - \frac{1}{4} \sin(2x)$$

$$\int \sin^n x dx = -\frac{\sin^{n-1} x \cos x}{n} + \frac{n-1}{n} \int \sin^{n-2} x dx$$

$$\int \cos^2 x dx = \frac{x}{2} + \frac{1}{4} \sin(2x)$$

$$\int \cos^n x dx = \frac{1}{n} \cos^{n-1} x \sin x + \frac{n-1}{n} \int \cos^{n-2} x dx$$

$$\int \sin x \cos x dx = \frac{1}{2} \sin^2 x$$

$$\int \sin^2 x \cos^2 x dx = -\frac{1}{8} \left[\frac{1}{4} \sin(4x) - x \right]$$

$$\int \sin x \cos^m x dx = -\frac{\cos^{m+1} x}{m+1}$$

$$\int \sin^m x \cos x dx = \frac{\sin^{m+1} x}{m+1}$$

$$\int \cos^m x \sin^n x dx = \frac{\cos^{m-1} x \sin^{n+1} x}{m+n} + \frac{m-1}{m+n} \int \cos^{m-2} x \sin^n x dx$$

$$\int \cos^m x \sin^n x dx = - \frac{\sin^{n-1} x \cos^{m+1} x}{m+n} + \frac{n-1}{m+n} \int \cos^m x \sin^{n-2} x dx$$

$$\int \frac{\sin^m x}{\cos^n x} dx = \frac{\sin^{m+1} x}{(n-1) \cos^{n-1} x} + \frac{n-m-2}{n-1} \int \frac{\sin^m x}{\cos^{n-2} x} dx$$

$$\int \frac{\cos^n x}{\sin^m x} dx = - \frac{\cos^{n+1} x}{(m-1) \sin^{m-1} x} + \frac{m-n-2}{m-1} \int \frac{\cos^n x}{\sin^{m-2} x} dx$$

$$\int \frac{dx}{\sin^m x} = - \frac{\cos x}{(m-1) \sin^{m-1} x} + \frac{m-2}{m-1} \int \frac{dx}{\sin^{m-2} x}$$

$$\int \frac{dx}{\cos^n x} = \frac{\sin x}{(n-1) \cos^{n-1} x} + \frac{n-2}{n-1} \int \frac{dx}{\cos^{n-2} x}$$

$$\int \tan x dx = - \log \cos x$$

$$\int \tan^2 x dx = \tan x - x$$

$$\int \cot x dx = \log \sin x$$

$$\int \cot^2 x dx = - \cot x - x$$

$$\int \sec x dx = \log \tan \left(\frac{\pi}{4} + \frac{x}{2} \right) = \frac{1}{2} \log \frac{1 + \sin x}{1 - \sin x}$$

$$\int \sec^2 x dx = \tan x$$

$$\int \operatorname{cosec} x dx = \log \tan \left(\frac{1}{2} x \right)$$

$$\int \operatorname{cosec}^2 x \, dx = -\cot x$$

$$\int \frac{dx}{1 + \sin ax} = -\frac{1}{a} \tan \left(\frac{\pi}{4} - \frac{ax}{2} \right)$$

$$\int \frac{dx}{1 - \sin ax} = \frac{1}{a} \cot \left(\frac{\pi}{4} - \frac{ax}{2} \right)$$

$$\int \frac{dx}{b+c \sin ax} = \frac{-2}{a\sqrt{b^2-c^2}} \tan^{-1} \left[\sqrt{\frac{b-c}{b+c}} \tan \left(\frac{\pi}{4} - \frac{ax}{2} \right) \right]$$

$b^2 > c^2$

$$\int \frac{dx}{b+c \sin ax} = -\frac{1}{a\sqrt{c^2-b^2}} \log \left(\frac{c+b \sin ax + \sqrt{c^2-b^2} \cos ax}{b+c \sin ax} \right)$$

$c^2 > b^2$

$$\int \sin ax \sin bx \, dx = \frac{\sin (a-b)x}{2(a-b)} - \frac{\sin (a+b)x}{2(a+b)}$$

$(a^2 \neq b^2)$

$$\int \frac{dx}{1 + \cos ax} = \frac{1}{a} \tan \frac{ax}{2}$$

$$\int \frac{dx}{1 - \cos ax} = -\frac{1}{a} \cot \frac{ax}{2}$$

$$\int \frac{dx}{b+c \cos ax} = \frac{2}{a\sqrt{b^2-c^2}} \tan^{-1} \left(\sqrt{\frac{b-c}{b+c}} \tan \frac{ax}{2} \right)$$

$b^2 > c^2$

$$\int \frac{dx}{b+c \cos ax} = \frac{1}{a\sqrt{c^2-b^2}} \log \left(\frac{c+b \cos ax + \sqrt{c^2-b^2} \sin ax}{b+c \cos ax} \right)$$

$c^2 > b^2$

$$\int \cos ax \cos bx \, dx = \frac{\sin (a-b)x}{2(a-b)} + \frac{\sin (a+b)x}{2(a+b)}$$

$a^2 \neq b^2$

$$\int \sin ax \cos bx \, dx = -\frac{1}{2} \left[\frac{\cos (a-b)x}{a-b} + \frac{\cos (a+b)x}{a+b} \right]$$

$a \neq b^2$

$$\int \frac{dx}{b \sin ax + c \cos ax} = \frac{1}{a \sqrt{b^2 + c^2}} \log \left[\tan \frac{1}{2} \left(ax + \tan^{-1} \frac{c}{b} \right) \right]$$

$$\int x \sin x \, dx = \sin x - x \cos x$$

$$\int x^2 \sin x \, dx = 2x \sin x - (x^2 - 2) \cos x$$

$$\int x \cos x \, dx = \cos x + x \sin x$$

$$\int x^2 \cos x \, dx = 2x \cos x + (x^2 - 2) \sin x$$

$$\int \frac{\sin ax \, dx}{x} = ax - \frac{(ax)^3}{3 \underline{3}} + \frac{(ax)^5}{5 \underline{5}} - \dots$$

$$\int \frac{\cos ax \, dx}{x} = \log ax - \frac{(ax)^2}{2 \underline{2}} + \frac{(ax)^4}{4 \underline{4}} - \dots$$

Transcendentals

$$\int \log x \, dx = x \log x - x$$

$$\int \frac{(\log x)^n}{x} \, dx = \frac{1}{n+1} (\log x)^{n+1}$$

$$\int \frac{dx}{x \log x} = \log \log x$$

$$\int \frac{dx}{x (\log x)^n} = -\frac{1}{(n-1) (\log x)^{n-1}}$$

$$\int x^m \log x \, dx = x^{m+1} \left[\frac{\log x}{m+1} - \frac{1}{(m+1)^2} \right]$$

$$\int x e^{ax} dx = \frac{e^{ax}}{a^2} (ax - 1)$$

$$\int x^m e^{ax} dx = \frac{x^m e^{ax}}{a} - \frac{m}{a} \int x^{m-1} e^{ax} dx$$

$$\int \frac{e^{ax}}{x^m} dx = -\frac{1}{m-1} \frac{e^{ax}}{x^{m-1}} + \frac{a}{m-1} \int \frac{e^{ax}}{x^{m-1}} dx$$

$$\int e^{ax} \sin (nx) dx = e^{ax} \left[\frac{a \sin (nx) - n \cos (nx)}{a^2 + n^2} \right]$$

$$\int e^{ax} \cos (nx) dx = e^{ax} \left[\frac{a \cos (nx) + n \sin (nx)}{a^2 + n^2} \right]$$

HYPERBOLIC FUNCTIONS

Hyperbolic Transformations

$$\sinh x = \frac{e^x - e^{-x}}{2} = -j \sin (jx)$$

where

$$j = \sqrt{-1}$$

$$\cosh x = \frac{e^x + e^{-x}}{2} = \cos (jx)$$

$$\tanh x = \frac{e^x - e^{-x}}{e^x + e^{-x}} = -j \tan (jx)$$

$$\coth x = \frac{e^x + e^{-x}}{e^x - e^{-x}} = j \cot (jx)$$

$$e^x = \cosh x + \sinh x$$

$$e^{-x} = \cosh x - \sinh x$$

$$\sin x = -j \sinh (jx)$$

$$\cos x = \cosh (jx)$$

Hyperbolic Formulæ

$$\cosh^2 x - \sinh^2 x = 1$$

$$\operatorname{sech}^2 x + \tanh^2 x = 1$$

$$\coth^2 x - \operatorname{cosech}^2 x = 1$$

$$\sinh (x + y) = \sinh x \cosh y + \cosh x \sinh y$$

$$\cosh (x + y) = \cosh x \cosh y + \sinh x \sinh y$$

$$\sinh (x - y) = \sinh x \cosh y - \cosh x \sinh y$$

$$\cosh (x - y) = \cosh x \cosh y - \sinh x \sinh y$$

$$\tanh (x + y) = \frac{\tanh x + \tanh y}{1 + \tanh x \tanh y}$$

$$\coth (x + y) = \frac{\coth x \coth y + 1}{\coth y + \coth x}$$

$$\tanh (x - y) = \frac{\tanh x - \tanh y}{1 - \tanh x \tanh y}$$

$$\coth (x - y) = \frac{\coth x \coth y - 1}{\coth y - \coth x}$$

$$\sinh (2x) = 2 \sinh x \cosh x$$

$$\cosh (2x) = \cosh^2 x + \sinh^2 x$$

$$\tanh (2x) = \frac{2 \tanh x}{1 + \tanh^2 x}$$

$$\coth (2x) = \frac{\coth^2 x + 1}{2 \coth x}$$

$$\sinh \left(\frac{x}{2} \right) = \sqrt{\frac{\cosh x - 1}{2}}$$

$$\cosh \left(\frac{x}{2} \right) = \sqrt{\frac{\cosh x + 1}{2}}$$

$$\tanh \left(\frac{x}{2} \right) = \sqrt{\frac{\cosh x - 1}{\cosh x + 1}}$$

$$\coth \left(\frac{x}{2} \right) = \sqrt{\frac{\cosh x + 1}{\cosh x - 1}}$$

$$\sinh x + \sinh y = 2 \sinh \left(\frac{x + y}{2} \right) \cosh \left(\frac{x - y}{2} \right)$$

$$\sinh x - \sinh y = 2 \cosh \left(\frac{x+y}{2} \right) \sinh \left(\frac{x-y}{2} \right)$$

$$\cosh x + \cosh y = 2 \cosh \left(\frac{x+y}{2} \right) \cosh \left(\frac{x-y}{2} \right)$$

$$\cosh x - \cosh y = 2 \sinh \left(\frac{x+y}{2} \right) \sinh \left(\frac{x-y}{2} \right)$$

$$\sinh (3x) = 3 \sinh x + 4 \sinh^3 x$$

$$\cosh (3x) = -3 \cosh x + 4 \cosh^3 x$$

Inverse Hyperbolic Functions

$$\sinh^{-1} x = \log (x + \sqrt{1+x^2})$$

$$\cosh^{-1} x = \log (x + \sqrt{x^2-1})$$

$$\tanh^{-1} x = \frac{1}{2} \log \left[\frac{1+x}{1-x} \right]$$

$$\coth^{-1} x = \frac{1}{2} \log \left[\frac{x+1}{x-1} \right]$$

$$\operatorname{sech}^{-1} x = \log \left(\frac{1}{x} + \sqrt{\frac{1}{x^2}-1} \right)$$

$$\operatorname{cosech}^{-1} x = \log \left(\frac{1}{x} + \sqrt{\frac{1}{x^2}+1} \right)$$

Differentials of Hyperbolic Functions

$$d(\sinh x) = \cosh x \, dx$$

$$d(\cosh x) = \sinh x \, dx$$

$$d(\tanh x) = \operatorname{sech}^2 x \, dx$$

$$d(\coth x) = -\operatorname{cosech}^2 x \, dx$$

$$d(\operatorname{sech} x) = -\operatorname{sech} x \tanh x \, dx$$

$$d(\operatorname{cosech} x) = -\operatorname{cosech} x \coth x \, dx$$

$$d(\sinh^{-1} x) = \frac{dx}{\sqrt{1+x^2}}$$

$$d(\cosh^{-1} x) = \frac{dx}{\sqrt{x^2-1}}$$

$$d(\tanh^{-1} x) = \frac{dx}{1-x^2}$$

$$d(\coth^{-1} x) = \frac{dx}{1-x^2}$$

$$d(\operatorname{sech}^{-1} x) = -\frac{dx}{x\sqrt{1-x^2}}$$

$$d(\operatorname{cosech}^{-1} x) = -\frac{dx}{x\sqrt{x^2+1}}$$

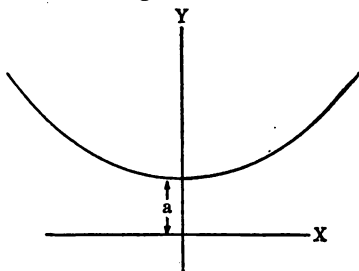
Use of Hyperbolic Functions

Illustrative Example. Deduce an expression for the length of a perfectly flexible chain suspended between two supports; assume that both points of support are the same height from the ground.

The chain assumes the form of a catenary (see page 33), the equation of which is

$$y = a \cosh \frac{x}{a}$$

The general equation for the length of the chain is



$$L = \text{length} = \int \sqrt{1 + \left(\frac{dy}{dx}\right)^2} dx$$

where the value of $\frac{dy}{dx}$, obtained by differentiating the equation of the catenary, is

$$\frac{dy}{dx} = \frac{d\left(a \cosh \frac{x}{a}\right)}{dx} = a \left[\left(\sinh \frac{x}{a} \right) \left(\frac{1}{a} \right) \right] = \sinh \frac{x}{a}$$

Substituting the value of $\frac{dy}{dx}$ in the formula for the length, L , we have

$$\begin{aligned} L &= \int \sqrt{1 + \sinh^2 \frac{x}{a}} dx = \int \sqrt{\cosh^2 \frac{x}{a}} dx \\ &= \int \cosh \frac{x}{a} dx = a \sinh \frac{x}{a} \end{aligned}$$

which is the required expression for the length of the chain.

DIFFERENTIAL EQUATIONS

A **differential equation** is a relation involving derivatives or differentials.

A **solution** of a differential equation is a relation between the variables which satisfies the given equation.

ORDINARY DIFFERENTIAL EQUATIONS

Equations of the First Order and First Degree

I. An equation of the form

$$f_1(x) dx + f_2(y) dy = 0$$

can be integrated immediately.

Its solution is

$$\int f_1(x) dx + \int f_2(y) dy = C$$

An equation may sometimes be changed to the above form by separation of the variables.

II. **Homogeneous Equation.** An equation is **homogeneous** in respect to its variables when the

sum of their exponents is the same for each term of the equation.

Homogeneous equations are reduced to the form of Method I, by substituting vx for y , and then separating the variables.

III. Non-homogeneous Equation of First Degree in x and y . This type occurs in the form:

$$(ax + by + c) dx = (a'x + b'y + c') dy$$

Substitute for x , $(x' + h)$, and for y , $(y' + k)$. The equation then becomes:

$$(ax' + by' + ah + bk + c) dx' = (a'x' + b'y' + a'h + b'k + c') dy'$$

Equating $ah + bk + c = 0$

and $a'h + b'k + c' = 0$

the original equation now takes the form:

$$(ax' + by') dx' = (a'x' + b'y') dy'$$

which is homogeneous and solvable by Method II.

In the solution thus obtained, substitute

$$x' = x - h \quad \text{and} \quad y' = y - k$$

where h and k are determined from the two equations:

$$ah + bk + c = 0$$

$$a'h + b'k + c' = 0$$

IV. Linear Equation. A linear differential equation (of first order and first degree) is of the general form:

$$\frac{dy}{dx} + Py = Q$$

where P and Q are functions of x alone or constants.

The solution of this equation is

$$ye^{\int P dx} = \int e^{\int P dx} Q dx + C$$

V. Equations Reducible to the Linear Equation. This type occurs in the form:

$$\frac{dy}{dx} + Py = Qy^n$$

where P and Q are functions of x alone. The given equation may be written:

$$\frac{dv}{dx} + (1 - n)Pv = (1 - n)Q$$

where $v = y^{-n+1}$. This equation is linear in v , and solvable by Method IV. In the solution, resubstitute for v its value y^{-n+1} .

VI. Exact Differential Equation. An equation of the form

$$M dx + N dy = 0$$

is exact if the derivative of M with regard to y is equal to the derivative of N with regard to x . The solution then is:

$$\int M dx + \int \left[N - \frac{\partial}{\partial y} \int M dx \right] dy = C$$

where $\int M dx$ is the integral of M with respect to x (regarding y as constant), and the term

$$\left[N - \frac{\partial}{\partial y} \int M dx \right]$$

is found by subtracting from N the derivative in respect to y of $\int M dx$. The term $\left[N - \frac{\partial}{\partial y} \int M dx \right]$ is integrated with regard to y (considering x constant). The complete solution is then given by the formula above.

VII. **Integrating Factors.** If a differential equation of the form

$$M dx + N dy = 0$$

is multiplied through by a certain expression called an integrating factor, the equation will become exact. It is then solvable by Method VI.

(a) When an equation is homogeneous, $\frac{1}{Mx + Ny}$ is an integrating factor.

(b) When the condition exists that

$$\frac{\frac{dM}{dy} - \frac{dN}{dx}}{N} = F(x) \quad [\text{an expression containing only } x]$$

then $e^{\int F(x) dx}$ is an integrating factor.

(c) Similarly when

$$\frac{\frac{dN}{dx} - \frac{dM}{dy}}{M} = F(y)$$

then $e^{\int F(y) dy}$ is an integrating factor.

Equations of the First Order but Higher than the First Degree

In the following formulæ, $\frac{dy}{dx}$ will be denoted by p .

An equation of first order and of n th degree is of the general form

$$p^n + Ap^{n-1} + Bp^{n-2} + \dots + Jp + K = 0$$

where the coefficients A, B, \dots, J, K are functions of x and y .

I. Clairaut's Equation. When an equation is of the form

$$y = px + f(p)$$

the solution is obtained by substituting for p a constant c ,

$$y = cx + f(c)$$

II. Solution by Factoring. The given equation may sometimes be resolved into rational factors of the first degree. Each factor is equated separately to zero, and its solution found by one of the preceding methods, using the same constant of integration in each case. The complete solution is then the product of the separate solutions.

III. Equations Containing only x and p . When an equation is of this type, solve for p , and substitute its value $\frac{dy}{dx}$. The resulting equation can be integrated immediately.

IV. Equations Containing only y and p . Solve for p , and substitute its value $\frac{dy}{dx}$. This equation is immediately integrable.

V. Equations Involving x , y , and p . A solution can be obtained by one of the following methods:

(a) Solve for x in terms of y and p . Then differentiate in respect to y , remembering that $\frac{dx}{dy} = \frac{1}{p}$.

The solution of this equation, together with the given equation, constitutes the complete solution.

(b) Solve for y in terms of x and p . Differentiate with respect to x , and in place of $\frac{dy}{dx}$ substitute its value

p . The complete solution consists of the solution of this equation, together with the original equation.

(c) Solve for p , and replace it with its value $\frac{dy}{dx}$.

From this equation it may be possible to obtain a solution.

Linear Differential Equations with Constant Coefficients

A **linear differential equation** is of the first degree in the dependent variable and all of its derivatives.

The **particular integral** is the solution of the equation obtained without the introduction of constants of integration.

The **complementary function** is the solution obtained by temporarily equating to zero all those terms of the equation that do not contain the dependent variable or derivatives thereof.

The **complete solution** is the sum of the particular integral and the complementary function.

A linear equation with **constant coefficients** is of the form:

$$\frac{d^n y}{dx^n} + P \frac{d^{n-1} y}{dx^{n-1}} + Q \frac{d^{n-2} y}{dx^{n-2}} + \dots + Ry = X$$

where the coefficients P, Q, \dots, R are constants; and X is a function of x . Replacing $\frac{d}{dx}$ by the symbol D , the equation becomes

$$(D^n + PD^{n-1} + QD^{n-2} + \dots + R)y = X.$$

Case I. Method of Solution when $X = 0$. Write the given integral in its symbolic form, replacing $\frac{d}{dx}$

by D . Then solve this equation for D as if it were an ordinary algebraic quantity.

When the roots of the equation (i.e., the values of D) are real, the solution is

$$y = c_1 e^{m_1 x} + c_2 e^{m_2 x} + \dots$$

where c_1, c_2 , etc., are the constants of integration, and m_1, m_2 , etc., are the roots of the equation.

When two or more real roots of the equation are equal, the solution is

$$y = (c_1 + c_2 x + c_3 x^2 + \dots) e^{mx} + \dots$$

where m is the value of the repeated root, and c_1, c_2, c_3 , etc., are the constants of integration (introduced in the manner shown in the above equation) and equal in number to the number of times the root m is repeated.

When the equation has imaginary roots (which always occur in pairs) the solution is

$$y = e^{m_1 x} [A \cos(a_1 x) + B \sin(a_1 x)] \\ + e^{m_2 x} [C \cos(a_2 x) + D \sin(a_2 x)] + \dots$$

where A and B, C and D , etc., are the constants of integration, and $(m_1 \pm a_1 \sqrt{-1}), (m_2 \pm a_2 \sqrt{-1})$, etc., are the complex imaginary roots of the equation.

When two or more pairs of complex imaginary roots are equal, the solution is

$$y = [(c_1 + c_2 x + \dots) \cos(ax) \\ + (c_3 + c_4 x + \dots) \sin(ax)] e^{mx}$$

where $(m \pm a \sqrt{-1})$ is the repeated pair of complex imaginary roots.

Case II. Method of solution when X is not equal to zero. In this case, the complete solution is

the **sum** of the complementary function and the particular integral.

The **complementary function** is found by temporarily equating $X = 0$, and obtaining the solution by the method of **Case I**.

The **particular integral** is obtained as follows.

The given equation is of the general form:

$$(D^n + PD^{n-1} + QD^{n-2} + \dots + R)y = X$$

in which D is used in place of $\frac{d}{dx}$.

In symbolic notation, this equation may be expressed

$$f(D)y = X$$

The particular integral can then be written:

$$y = \frac{X}{f(D)} = \text{particular integral}$$

A. Method of obtaining the particular integral when the term X is of the form e^{ax} .

$$\text{particular integral} = \frac{X}{f(D)} = \frac{e^{ax}}{f(D)} = \frac{e^{ax}}{f(a)}$$

which is found by substituting the constant a in place of D .

This method for evaluating $\frac{e^{ax}}{f(D)}$ fails when the term $(D - a)$ is a factor of $f(D)$. The particular integral is then found by substituting the constant a for D in all terms of $f(D)$ except in the factor $(D - a)$. The solution is then completed by the general method given under case F (page 84).

B. Solution for the particular integral when X has the form x^m .

$$\text{particular integral} = \frac{X}{f(D)} = \frac{x^m}{f(D)} = [f(D)]^{-1} x^m$$

To evaluate this expression, expand $[f(D)]^{-1}$ into a series of ascending powers of D , by use of the binomial theorem. It is only necessary to carry out this expansion to the m th power of D , since operation on x^m by higher powers of D would produce zero (since the symbol D stands for $\frac{d}{dx}$, the operation by D on a quantity denotes its derivative with respect to x , the operation by D^2 denotes its second derivative, etc.). In obtaining the solution of the given particular integral, x^m is operated on separately by each term of the expansion of $[f(D)]^{-1}$.

C. Method of obtaining the particular integral when X has the form $\sin(ax)$.

$$\text{particular integral} = \frac{X}{f(D)} = \frac{\sin(ax)}{f(D)}$$

In order to evaluate this integral, substitute $-a^2$ for D^2 wherever D^2 occurs in $f(D)$. The particular integral will then be a fraction, whose numerator is $\sin(ax)$, and whose denominator is the value assumed by $f(D)$ when D^2 is replaced by $-a^2$.

This method fails if $f(D)$ becomes zero when $-a^2$ is substituted for D^2 . The particular integral is then evaluated by writing the term e^{iax} (in which $i = \sqrt{-1}$) in place of $\sin(ax)$. The solution of this new integral is obtained by method A for the evaluation of the particular integral. In the result, e^{iax} is replaced by $[\cos(ax) + i \sin(ax)]$, producing a result containing both real and imaginary terms. The required particular integral is the coefficient of i (i.e., $\sqrt{-1}$) in this expression.

D. Particular Integral when $X = \cos(ax)$. The

particular integral is obtained as in method C, with the exception that $\cos(ax)$ is used in place of $\sin(ax)$.

When this method fails, e^{iax} is written in place of $\cos(ax)$, and this new integral is evaluated by method A. In the solution of this integral, e^{iax} is replaced by $[\cos(ax) + i \sin(ax)]$. The required particular integral is the real part of this result.

E. Particular integral when X is of the form $e^{ax}Q$.

$$\text{particular integral} = \frac{X}{f(D)} = \frac{e^{ax}Q}{f(D)} = e^{ax} \frac{Q}{f(D+a)}$$

To evaluate the given integral, $(D+a)$ is substituted for D , wherever D occurs in $f(D)$; and the term e^{ax} is treated as a constant multiplier. The new integral

$\frac{Q}{f(D+a)}$ is evaluated by one of the preceding methods, or by the general method F. The required particular integral is then equal to the product of e^{ax} by the evaluation of $\frac{Q}{f(D+a)}$.

F. General method for finding the particular integral.

To evaluate $\frac{1}{f(D)} X$

The denominator of $\frac{1}{f(D)}$ may be resolved into factors of the first degree. The given integral then becomes:

$$\frac{1}{(D-a)} \frac{1}{(D-b)} \frac{1}{(D-c)} \frac{1}{(D-d)} \cdots \frac{1}{(D-m)} X$$

The term X is operated on successively by each of these fractional operators, beginning at the right. The

operation on X by the first factor $\frac{1}{(D - m)}$ produces the expression $e^{mx} \int e^{-mx} X dx$. This result is operated on in a similar manner by each remaining factor (proceeding from right to left). The solution of the given particular integral is then:

$$e^{ax} \int e^{-ax} e^{bx} \int e^{-bx} e^{cx} \int e^{-cx} \dots e^{mx} \int e^{-mx} X (dx)^m$$

Homogeneous Linear Equation

The homogeneous linear equation is of the form

$$x^n \frac{d^n y}{dx^n} + P x^{n-1} \frac{d^{n-1} y}{dx^{n-1}} + \dots + R y = X$$

in which the coefficients P, \dots, R are constants, and X is a function of x .

On assuming the relation, $x = e^z$, this equation may be transformed by the substitutions:

$$x^n \frac{d^n y}{dx^n} = \theta (\theta - 1) (\theta - 2) \dots \text{ to } n \text{ terms}$$

$$x^{n-1} \frac{d^{n-1} y}{dx^{n-1}} = (\theta - 1) (\theta - 2) (\theta - 3) \dots \text{ to } (n - 1) \text{ terms,}$$

and so forth; where the symbol θ stands for $\frac{d}{dz}$.

The **complementary function** is then found as in the case of the linear equation with constant coefficients. (In obtaining this solution, the term θ is treated in exactly the same manner in which the term D was treated in the preceding cases.)

In order to obtain the **particular integral**, the term X (which involves only x) is changed to an expression

involving z , by the substitution $x = e^z$. The particular integral is then found by one of the methods given under the case of the linear equation with constant coefficients.

The **complete solution** is the sum of the complementary function and the particular integral. In the result, z is replaced by its value $\log x$.

Exact Differential Equations

An exact differential equation is one which can be derived directly by differentiation of an equation of the next lower order.

If the given equation is of the form:

$$A \frac{d^n y}{dx^n} + B \frac{d^{n-1} y}{dx^{n-1}} + \dots + Q \frac{d^3 y}{dx^3} + R \frac{d^2 y}{dx^2} + S \frac{dy}{dx} + Ty = X$$

where A, B, \dots, Q, R, S, T , and X are functions of x , we then have as the condition for exactness that:

$$T - \frac{dS}{dx} + \frac{d^2 R}{dx^2} - \frac{d^3 Q}{dx^3} + \dots = 0$$

The first integral of the given equation then is:

$$A \frac{d^{n-1} y}{dx^{n-1}} + \left(B - \frac{dA}{dx} \right) \frac{d^{n-2} y}{dx^{n-2}} + \left(C - \frac{dB}{dx} + \frac{d^2 A}{dx^2} \right) \frac{d^{n-3} y}{dx^{n-3}} \dots = \int X dx + C$$

This formula may be reapplied successively as long as each resulting equation satisfies the condition for exactness.

Equations of the Second Order and the First Degree

General form is

$$\frac{d^2y}{dx^2} + P \frac{dy}{dx} + Qy = X$$

where P , Q , and X are functions of x .

I. When one solution of the equation is known (or can be found by inspection).

Let y_1 equal the known integral. In the given equation, substitute vy_1 in place of y ; and then, in the transformed equation, replace $\frac{dv}{dx}$ by p . This equation can be solved by one of the preceding methods.

II. Change of the Independent Variable.

The purpose of this change and of the removal of the first derivative (see III) is to transform a given equation into a new equation which may happen to be easily integrable.

The given equation is of the form:

$$\frac{d^2y}{dx^2} + P \frac{dy}{dx} + Qy = X$$

By changing the independent variable, it may be transformed into the following equation:

$$\frac{d^2y}{dz^2} + P_1 \frac{dy}{dz} + Q_1y = X_1$$

where Q_1 becomes equal to 1, if

$$\frac{dz}{dx} = \sqrt{Q}$$

when also

$$P_1 = \frac{\frac{d^2z}{dx^2} + P \frac{dz}{dx}}{Q}$$

and

$$X_1 = \frac{X}{Q}$$

or where P_1 may be made equal to zero, if

$$z = \int e^{-\int P dx} dx$$

when also
$$Q_1 = \frac{Q}{\left(\frac{dz}{dx}\right)^2}$$

and
$$X_1 = \frac{X}{\left(\frac{dz}{dx}\right)^2}$$

III. Removal of the First Derivative.

To remove the first derivative from an equation of

the form
$$\frac{d^2y}{dx^2} + P \frac{dy}{dx} + Qy = X$$

make the substitution $y = ve^{-\frac{1}{2}\int P dx}$

The given equation then becomes

$$\frac{d^2v}{dx^2} + Q_1v = X_1$$

where
$$Q_1 = Q - \frac{1}{2} \frac{dP}{dx} - \frac{1}{4} P^2$$

and
$$X_1 = Xe^{\frac{1}{2}\int P dx}$$

THEORETICAL MECHANICS

Center of Gravity

The center of gravity of a body is a point so situated that the force of gravity produces no tendency in the body to rotate about any axis passing through this point.

Center of Gravity of the Arc of a Plane Curve

$$\bar{x} = \frac{\int x ds}{\int ds} = \frac{\int x \sqrt{1 + \left(\frac{dy}{dx}\right)^2} dx}{\int \sqrt{1 + \left(\frac{dy}{dx}\right)^2} dx}$$

$$\bar{y} = \frac{\int y ds}{\int ds} = \frac{\int y \sqrt{1 + \left(\frac{dx}{dy}\right)^2} dy}{\int \sqrt{1 + \left(\frac{dx}{dy}\right)^2} dy}$$

where \bar{x} and \bar{y} are the coördinates of the center of gravity.

Solve for y in terms of x from the equation of the given curve. Then differentiate in order to obtain $\frac{dy}{dx}$, and substitute its value in the formula for \bar{x} .

Similarly, find x in terms of y , obtain $\frac{dx}{dy}$, and substitute in the formula for \bar{y} .

Center of Gravity of Plane Areas

Rectangular Coördinates:

$$\bar{x} = \frac{\int \int x dA}{\int \int dA} = \frac{\int \int x dx dy}{\int \int dx dy}$$

$$\bar{y} = \frac{\int \int y dA}{\int \int dA} = \frac{\int \int y dx dy}{\int \int dx dy}$$

where \bar{x} and \bar{y} are the coördinates of the center of gravity.

In evaluating the expression for \bar{x} , we may integrate first either in respect to x or y , according to which method is more convenient.

If dy is integrated first, the limits of y are expressed in terms of x (from the given equation); and the limits of x are its initial and final values.

Similarly, if dx is first integrated, the limits of x are expressed in terms of y ; and the limits of y are then its initial and final values.

Polar Coördinates:

$$\bar{x} = \frac{\int \int r^2 \cos \theta \, d\theta \, dr}{\int \int r \, d\theta \, dr}$$

$$\bar{y} = \frac{\int \int r^2 \sin \theta \, d\theta \, dr}{\int \int r \, d\theta \, dr}$$

Generally, it is more convenient to integrate first with respect to r . In this case, the limits of r are found in terms of θ from the equation of the given curve. The limits of θ are its initial and final values, expressed in radians.

Center of Gravity of Solids of Revolution. When a solid of uniform density is formed by the revolution of a plane curve about the X -axis, the center of gravity is on the X -axis (because of symmetry). Its x -coördinate is

$$\bar{x} = \frac{\int \int xy \, dx \, dy}{\int \int y \, dx \, dy}$$

where the limits are found as in the case of plane areas.

When a solid is formed by the revolution of a plane figure about the Y -axis, the y -coördinate of its center of gravity is

$$\bar{y} = \frac{\int \int xy \, dx \, dy}{\int \int x \, dx \, dy}$$

Center of Gravity of Any Section Composed of Two or More Simple Plane Figures

In order to find the center of gravity of such figures as tee-bars, channels, rails, etc., divide them up into their component rectangles or triangles. Then, obtain the center of gravity and the area of each separate figure. Choose any convenient axis in the plane of the given section and find the turning moment of each figure about this axis. Each turning moment is the product of the area of the figure by the distance from its center of gravity to the chosen axis. The sum of all these separate turning moments gives the turning moment of the total figure. On dividing this total moment by the total area of the figure, we obtain the distance from the chosen axis to the center of gravity of the figure. Care must be used, if the chosen axis passes through the given figure, to take distances on one side of this axis as positive, and on the other side as negative.

Generally, one coördinate of the center of gravity can be determined by the symmetry of the given section. When the figure is unsymmetrical, it may be necessary to take moments about two different axes in order to locate the center of gravity.

Moment of Inertia of Plane Areas

The **moment of inertia of a plane figure** about any given axis is equal to the integral of the product of each elementary area of the figure by the square of its distance from the axis.

Rectangular Moment of Inertia:

The **rectangular moment of inertia** of a plane figure

is its moment of inertia about any axis in the plane of the figure. The rectangular moment of inertia of a plane area about the X -axis is

$$I_x = \int \int y^2 dx dy$$

The rectangular moment of inertia of a plane area about the Y -axis is

$$I_y = \int \int x^2 dx dy$$

In either case, the limits of the variable first integrated are expressed in terms of the other variable.

The **moment of inertia** of a plane figure **about the gravity axis** (I_g) is its rectangular moment of inertia about any axis in the plane of the figure, passing through its center of gravity.

The **moment of inertia** of a plane figure **about any axis parallel to the gravity axis** and in the plane of the figure is equal to (I_g) plus the product of the area of the figure by the square of the distance between the two axes, thus:

$$I = I_g + Fd^2$$

Polar Moment of Inertia:

The **polar moment of inertia** (I_p) is the moment of inertia about any axis perpendicular to the plane of the given figure.

It is equal to the sum of the rectangular moments of inertia about two mutually perpendicular axes in the plane of the figure, passing through the foot of the polar axis.

In **rectangular coördinates**, the polar moment of inertia equals

$$I_p = I_x + I_y = \int \int (x^2 + y^2) dx dy$$

In **polar coördinates**, the formula for the polar moment of inertia is

$$I_p = \int \int R^3 dR d\theta$$

It is generally more convenient to integrate first with respect to R , expressing its limits in terms of θ . The limits of θ are then its initial and final values.

Moment of Inertia of Solids

The moment of inertia of a solid (with center at origin) about the X -axis is

$$I = m \int \int \int (y^2 + z^2) dx dy dz$$

where m is the density, that is, the mass per unit volume.

Radius of Gyration

The **center of gyration** is that point in a revolving body at which, if the entire mass of the body were concentrated, the moment of inertia about the axis of rotation would be the same as that of the body.

The **radius of gyration**, k , is the distance from the axis of rotation to the center of gyration.

$$\text{For plane sections, } k = \sqrt{\frac{I}{A}}$$

$$\text{For solids, } k = \sqrt{\frac{I}{M}} = \sqrt{\frac{I}{\left(\frac{W}{g}\right)}}$$

in which k = radius of gyration,

I = the moment of inertia about the axis of rotation,

A = area of section,

M = mass of body,

W = weight of body.

Center of Percussion

The **center of percussion** or oscillation of a pendulum or other body vibrating or rotating about a fixed axis or center is that point at which, if the entire weight of the body were concentrated, the body would continue to vibrate in the same intervals of time.

The **radius of oscillation** is

$$h = \frac{I}{Md} = \frac{I}{\left(\frac{W}{g}\right)d}$$

in which I = the moment of inertia of body about axis of rotation,

d = distance from center of gravity of body to the axis of rotation,

h = distance from center of percussion or oscillation to the axis of rotation,

M = mass of body,

W = weight of body.

Motion of a Body

$$\text{velocity at any instant} = v = \frac{ds}{dt}$$

$$\text{acceleration at any instant} = a = \frac{dv}{dt} = \frac{d^2s}{dt^2}$$

In rectangular coördinates,

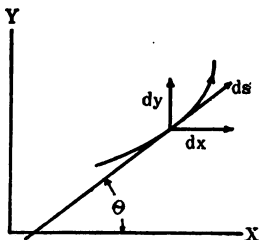
$$v_x = \frac{dx}{dt} = \frac{ds}{dt} \cos \theta = \text{velocity in a direction parallel to the } X\text{-axis}$$

$$v_y = \frac{dy}{dt} = \frac{ds}{dt} \sin \theta$$

$$v = \frac{ds}{dt} = \sqrt{\left(\frac{dx}{dt}\right)^2 + \left(\frac{dy}{dt}\right)^2}$$

For motion with **uniform velocity**,

$$v = \frac{s}{t}$$



For **uniformly accelerated motion**,

$$s = \frac{1}{2} (u + v) t$$

$$s = ut + \frac{1}{2} at^2$$

$$2as = v^2 - u^2$$

u = initial velocity,

v = final velocity,

a = constant acceleration,

s = space passed over,

t = time of motion.

If the **body starts from rest**, the initial velocity u equals 0, and these equations become:

$$s = \frac{1}{2} vt$$

$$s = \frac{1}{2} at^2$$

$$2as = v^2$$

Rotation of a Rigid Body

$$\text{velocity at any instant} = \omega = \frac{d\theta}{dt}$$

$$\text{acceleration at any instant} = \alpha = \frac{d\omega}{dt} = \frac{d^2\theta}{dt^2}$$

For motion with **uniform velocity**,

$$\omega = \frac{\theta}{t}$$

For **uniformly accelerated motion**,

$$\theta = \frac{1}{2} (\omega_0 + \omega) t$$

$$\theta = \omega_0 t + \frac{1}{2} \alpha t^2$$

$$2 \alpha \theta = \omega^2 - \omega_0^2$$

θ = angular space through which the body rotates,

ω_0 = initial angular velocity,

ω = final angular velocity,

α = angular acceleration,

t = time.

For a **body initially at rest**, the velocity ω_0 is 0, and these equations become

$$\theta = \frac{1}{2} \omega t$$

$$\theta = \frac{1}{2} \alpha t^2$$

$$2 \alpha \theta = \omega^2$$

Falling Bodies

Equations of motion of a **body falling from rest** under the action of gravity:

$$v = gt$$

$$s = \frac{1}{2} gt^2$$

$$2 gs = v^2$$

v = velocity after time t ,

s = height through which body falls,

g = (approx.) 32.16 feet/sec.² = 981 cm/sec.²
= acceleration of gravity.

The value of g for any latitude and any altitude is

$$g = 32.0894 (1 + 0.0052375 \sin^2 \theta) \\ \times (1 - 0.0000000957 E)$$

in which

θ = latitude of place in degrees,

E = elevation above sea-level in feet.

Projectiles

Equations of a body projected vertically upward with an initial velocity u (resistance of air not considered):

(1) Velocity at any time = $u - gt$.

(2) Velocity at any height = $\sqrt{u^2 - 2gh}$.

(3) Height at any time = $ut - \frac{1}{2}gt^2$.

(4) Greatest height = $\frac{u^2}{2g}$.

(5) Time of flight = $\frac{2u}{g}$.

Equations of a body projected with an initial velocity u at an angle θ° to the horizontal (resistance of air not considered):

The curve described by the projectile is the parabola whose equation is

$$y = x \tan \theta - \frac{gx^2}{2u^2 \cos^2 \theta}$$

where θ is positive when the body is projected above

the horizontal and negative when the body is projected below the horizontal.

$$\text{Horizontal-component of acceleration} = \frac{d^2x}{dt^2} = 0$$

$$\text{Vertical-component of acceleration} = \frac{d^2y}{dt^2} = -g$$

$$(1) \text{ Velocity at any time} = \sqrt{u^2 - 2utg \sin \theta + g^2t^2}$$

$$(2) \text{ Velocity at any height} = \sqrt{u^2 - 2gh}$$

$$(3) \text{ Height at any time} = ut \sin \theta - \frac{1}{2}gt^2$$

$$(4) \text{ Time of flight} = \frac{2u \sin \theta}{g}$$

$$(5) \text{ Range} = \frac{u^2 \sin (2\theta)}{g}$$

If the friction of the air is taken into account, the curve described by the projectile is given by the empirical relation:

$$y = x \tan \theta - \frac{gx^2}{2 \cos^2 \theta} \left(\frac{1}{u^2} + \frac{kx}{u} \right)$$

$$k = 0.0000000458 \frac{d^2}{w}$$

where d = diameter of projectile in inches.

w = weight of projectile in pounds.

Angular Measure

A **radian** is the angle subtended at the center of any circle by an arc equal in length to its radius.

$$1 \text{ radian} = \frac{180}{\pi} \text{ degrees} = 57.296+ \text{ degrees}$$

$$1 \text{ degree} = \frac{\pi}{180} \text{ radian} = 0.017453+ \text{ radian}$$

The relation between the central angle of a circle and its subtended arc is given by the formula:

$$l = r\theta$$

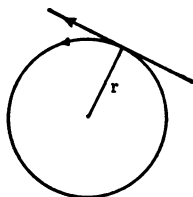
- l = length of arc,
 r = radius of circle,
 θ = central angle in radians.

Circular Motion

A body moving with **uniform velocity** in a circular path experiences a constant acceleration toward the center of the circle. This acceleration is expended in changing the direction of motion of the body.

The equations of motion of the revolving body are

$$\begin{aligned}
 a &= \frac{v^2}{r} \\
 vT &= 2\pi r \\
 a &= \frac{4\pi^2 r}{T^2}
 \end{aligned}$$



v = constant velocity of particle in feet per second,
 a = constant acceleration toward center in feet per sec.²,

r = radius of circular path in feet,
 T = time of 1 revolution in seconds,
 $\pi^2 = 9.8696 +$.

If the body moves with a **variable velocity**, then:

$$\text{tangential acceleration} = \frac{dv}{dt}$$

$$\text{normal acceleration} = \frac{v^2}{r}$$

Centrifugal Force

The centrifugal force of a revolving body, in pounds, is

$$F = \frac{Wv^2}{gr} = \frac{4\pi^2 W r}{gt^2}$$

or in terms of the number of revolutions, N_1 , per minute

$$F = 0.00034 W r N_1^2$$

W = weight of revolving body in pounds,

v = velocity of body in feet per second,

t = time of 1 revolution in seconds,

r = distance from axis of rotation to the center of gravity of the body, in feet,

g = acceleration of gravity (32.16).

Flywheel

The energy of rotation of a flywheel is

$$K.E. = \frac{I\omega^2}{2} = 2\pi^2 I N^2$$

I = polar moment of inertia about the axis of rotation,

ω = angular velocity in radians per second,

N = number of revolutions per second.

The energy stored in a rim flywheel by a variation in speed is

$$E = \frac{W}{2g} (S_{\max}^2 - S_{\min}^2) \text{ foot-pounds,}$$

W = weight of flywheel in pounds,

S_{\max} = maximum rim speed in feet per second,

S_{\min} = minimum rim speed in feet per second

g = acceleration of gravity (32.16).

The rim speed in feet per second is $S = 2\pi RN$, where N is the speed in revolutions per second, and R is the radius of the wheel in feet, measured from the center of gravity of the rim section.*

* This value of R is approximately correct. The exact value of R is the radius of gyration of the flywheel.

Hence, the **energy stored** is

$$E = \frac{W}{g} \frac{4 \pi^2 R^2 (N_{\max}^2 - N_{\min}^2)}{2} \text{ foot-pounds}$$

and the **weight** of the flywheel is

$$W = \frac{Eg}{2 \pi^2 R^2 (N_{\max}^2 - N_{\min}^2)}$$

Substitute for E the required stored energy in foot-pounds. Assume some convenient value for R , in feet; then solve for the weight W in pounds. If the rim speed is too high (average about 35 feet per second for cast iron or 150 feet per second for steel), the value of R must be reduced. The ratio of the speed variation, $N_{\max} - N_{\min}$, to the average speed may be taken as follows for different types of machines:

| | |
|----------------------------------|-------|
| Hammers..... | 0.20 |
| Punches..... | 0.05 |
| Ordinary machinery..... | 0.03 |
| Textile and paper machinery..... | 0.02 |
| Electric generators..... | 0.005 |

* This value of R is approximately correct. The exact value of R is the radius of gyration of the flywheel.

Simple Pendulum

The **time of oscillation** in seconds from one extreme position to the other is

$$t = \pi \sqrt{\frac{l}{g}}$$

l = length of pendulum in feet,

g = acceleration of gravity (32.16 approx.).

The **period** of the pendulum is

$$P = 2t = 2\pi \sqrt{\frac{l}{g}}$$

The **seconds-pendulum** makes one oscillation per

second from one extreme position to the other; its length in feet is

$$l = \frac{g}{\pi^2}$$

Work and Energy

For a **uniform** force,

$$F = ma = \frac{W}{g} a$$

$$Ft = mv = \frac{W}{g} v$$

$$Fs = \frac{1}{2} mv^2 = \frac{Wv^2}{2g}$$

F = constant applied force in pounds,

a = constant acceleration in feet/sec.²,

m = mass of body,

W = weight of body in pounds,

v = velocity acquired after t seconds,

mv = momentum,

s = space passed over in feet,

g = acceleration of gravity (32.16 feet/sec.²).

The **impulse** I of the constant force F during the time t equals the change of momentum,

$$I = Ft = mv - mu$$

where u is the initial velocity and v the final velocity.

If the force is variable, then **impulse** equals

$$I = \int_0^t F dt$$

The **work** done by a uniform force is

$$W = Fs = \frac{1}{2} mv^2$$

The work done by a variable force equals

$$W = \int_0^s F ds$$

The kinetic energy of a body of mass m , moving with a velocity v , equals $\frac{1}{2} mv^2$.

Direct Central Impact

For the impact of two bodies of the same material, weighing respectively W and W_1 pounds, the velocities after impact are

$$v = \frac{Wu + W_1u_1 - eW_1(u - u_1)}{W + W_1}$$

$$v_1 = \frac{Wu + W_1u_1 + eW(u - u_1)}{W + W_1}$$

u = original velocity of W in feet/second,

v = velocity of W after impact,

u_1 = original velocity of W_1 ,

v_1 = velocity of W_1 after impact,

e = coefficient of restitution.

Values of e , the coefficient of restitution, for different materials are as follows:

| | |
|-----------------------------|------------|
| glass on glass..... | $e = 0.94$ |
| ivory on ivory..... | $e = 0.81$ |
| cast iron on cast iron..... | $e = 0.66$ |
| lead on lead..... | $e = 0.2$ |

The sum of the momenta of two bodies after impact equals the sum of their momenta before impact,

$$\frac{Wv}{g} + \frac{W_1v_1}{g} = \frac{Wu}{g} + \frac{W_1u_1}{g}$$

Two **inelastic** bodies after impact move with a common velocity

$$v = \frac{W_1 v_1 + W_2 v_2}{W_1 + W_2}$$

in which

W_1 = weight of first body,

W_2 = weight of second body,

v_1 = original velocity of first body,

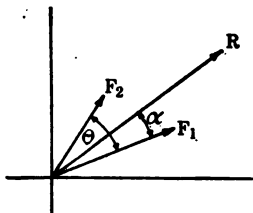
v_2 = original velocity of second body.

Composition and Resolution of Forces

The **resultant** of the forces F_1 and F_2 acting at a point is

$$R = \sqrt{F_1^2 + 2 F_1 F_2 \cos \theta + F_2^2}$$

in which θ is the angle in degrees between the two forces.



The **direction** of R is determined by the relation

$$\tan \alpha = \frac{F_2 \sin \theta}{F_1 + F_2 \cos \theta}$$

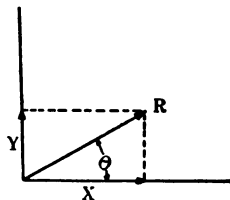
in which α is the angle in degrees between F_1 and R .

The **rectangular components** of a force R acting in a given direction are

$$X = R \cos \theta$$

$$Y = R \sin \theta$$

in which X is the horizontal component of R , Y is the normal component of R , and θ is the angle in degrees between R and X .



The **resultant** of several forces acting in different directions at a point is

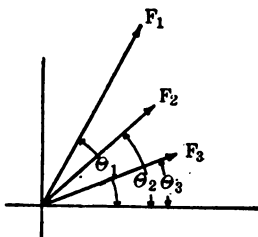
$$R = \sqrt{X^2 + Y^2}$$

in which

$$X = F_1 \cos \theta_1 + F_2 \cos \theta_2 + F_3 \cos \theta_3 + \dots,$$

$$Y = F_1 \sin \theta_1 + F_2 \sin \theta_2 + F_3 \sin \theta_3 + \dots,$$

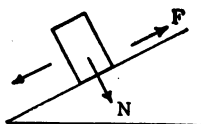
where F_1, F_2, F_3 , etc., are the given forces, and $\theta_1, \theta_2, \theta_3$, etc., are the angles in degrees between the given forces and the horizontal axis.



Friction

F = friction in pounds,
 N = normal force in pounds,
 f = coefficient of friction.

$$F = fN$$



$$\text{Angle of friction} = \phi = \tan^{-1} \frac{F}{N} = \tan^{-1} f$$

Average values for f , the coefficient of friction, for motion are as follows:

| Character of contact | f |
|-----------------------------------|-----------|
| Wood on wood..... | 0.25-0.50 |
| Metal on wood..... | 0.50-0.60 |
| Metal on metal, dry..... | 0.15-0.24 |
| Metal on metal, lubricated..... | 0.075 |
| Leather on metal, dry..... | 0.56 |
| Leather on metal, lubricated..... | 0.15 |

Belt Friction

P and Q are the forces at the ends of the belt, P being the greater force.

F = resultant force of friction,

N = normal reaction of pulley,

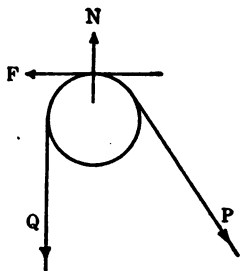
θ = angle in radians subtended by the arc of contact,

f = coefficient of friction.

$$\log_e \frac{P}{Q} = f\theta$$

or in common logarithms

$$\log_{10} \frac{P}{Q} = 0.434 f\theta$$



The value of f varies from 0.15 to 0.6 depending on the condition of belt and pulley, but, in general, it is approximately correct to assume $f = 0.3$.

Inclined Plane

Equations of motion of a body sliding down an incline under the action of its own weight.

For a frictionless plane:

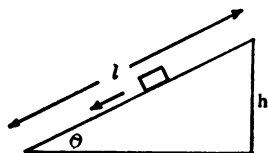
(1) acceleration along plane = $a = \frac{d^2s}{dt^2} = g \sin \theta$,

(2) velocity after t seconds = $tg \sin \theta$,

(3) velocity at bottom of plane = $\sqrt{2gh}$,

(4) distance traveled in t seconds = $\frac{t^2 g \sin \theta}{2}$,

(5) time of sliding down plane = $t \sqrt{\frac{2}{gh}}$.



For an inclined plane with friction:

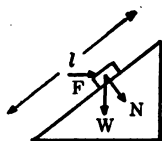
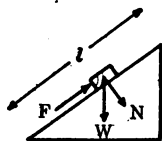
$$(1) \text{ acceleration along plane} = a = \frac{d^2s}{dt^2} \\ = g [\sin \theta - f \cos \theta],$$

in which

f = coefficient of friction.

Conditions for the equilibrium of a body resting on an incline:

- W = weight of body,
- F = applied force,
- N = normal pressure on plane,
- θ = inclination of plane in degrees,
- f = coefficient of friction.



For a frictionless plane:

(1) When the balancing force is applied parallel to the inclined plane,

$$F = W \sin \theta \\ N = W \cos \theta$$

(2) When the applied force acts horizontally,

$$F = W \tan \theta, \\ N = W \sec \theta.$$

For an inclined plane with friction:

(1) When the balancing force acts parallel to the incline,

$$F = \frac{W \sin (\theta \pm \theta')}{\cos (\theta')}$$

in which

$$\theta' = \tan^{-1} f$$

(2) When the applied force acts horizontally,

$$F = W \tan (\theta \pm \theta')$$

MECHANICS OF MATERIALS

Stress is distributed force; its intensity per unit area is generally expressed in pounds per square inch.

The **elastic limit** of a material is the maximum stress in pounds per square inch that will be followed by a complete recovery of form, after the removal of the stress.

Permanent set is the change in form of a member when stressed beyond its elastic limit.

The **ultimate strength** of a material is the least stress in pounds per square inch that will produce rupture.

Modulus of elasticity is the number obtained by dividing the actual stress in pounds per square inch by the corresponding elongation per inch.

The **factor of safety** is the factor obtained by dividing the ultimate strength by the actual stress in pounds per square inch.

Tension and Compression

For direct stress, uniformly distributed,

$$p = \frac{P}{F}$$

p = stress in pounds per square inch,

P = total load in pounds,

F = cross-sectional area in square inches.

$$E = \frac{p}{\epsilon} \quad \epsilon = \frac{\lambda}{l}$$

$$E = \frac{P}{\frac{F}{\lambda}} = \frac{Pl}{F\lambda}$$

E = modulus of elasticity in tension or compression,

l = length of member in inches,

ϵ = elongation per inch length,

λ = total elongation in inches.

STRENGTH OF MATERIALS

| Material | Density | Elastic limit | Ultimate strength | | | Modulus of elasticity | | Factor of safety | | | |
|--------------------------|---------|---------------|-------------------|---------|--------|-----------------------|------------|------------------|-----------|--------|-------|
| | | | Tension | Comp. | Shear | Tens. and comp. | Shear | Steady load | Var. load | Shocks | |
| Brick..... | 2 | | | 3,000 | 1,000 | | 2,000,000 | | 15 | 25 | 40 |
| Stone..... | 2.6 | | | 6,000 | 1,500 | | 6,000,000 | | 15 | 25 | 40 |
| Timber..... | 0.6 | 3,000 | 10,000 | 8,000 | | | 1,500,000 | | 8 | 10 | 15 |
| Timber along grain..... | 0.6 | | | | 500 | | | | | | |
| Timber across grain..... | 0.6 | | | | 3,000 | | | 400,000 | | | |
| Cast iron..... | 7.2 | 6,000 | 20,000 | 90,000 | 18,000 | | 15,000,000 | 6,000,000 | 6 | 10 | 20 |
| Wrought iron..... | 7.7 | 25,000 | 50,000 | 50,000 | 40,000 | | 25,000,000 | 10,000,000 | 4 | 6 | 10 |
| Structural steel.. | 7.8 | 35,000 | 60,000 | 60,000 | 50,000 | | 30,000,000 | 12,000,000 | 4 | 6 | 10 |
| Strong steel..... | 7.8 | 50,000 | 100,000 | 120,000 | 80,000 | | 30,000,000 | 12,000,000 | 5 | 8 | 15 |

Note.—The elastic limit of 6,000 for cast iron holds only for tension; for compression, the elastic limit is 20,000.

Angular Distortion and Shear

Shearing stress, uniformly distributed equals

$$p_s = \frac{P}{F}$$

P = load,

F = area.

For torsion:

$$E_s = \frac{p_s}{\delta}$$

E_s = modulus of elasticity in shear,

δ = angle of distortion in radians.

Note. The modulus of elasticity in shear is $\frac{2}{3}$ as great as in compression or tension.

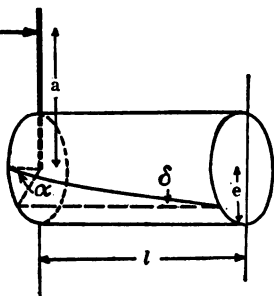
Torsion of Circular Shafts

$$\delta = \frac{e\alpha}{l} \quad p_s = \frac{e\alpha E_s}{l} P$$

$$Pa = \frac{p_s I_p}{e} = \frac{\alpha I_p E_s}{l}$$

$$I_p = \frac{\pi d^4}{32}$$

$$Pa = \frac{\pi p_s d^3}{16}$$



$$\text{Horsepower} = \frac{2\pi PaN}{33,000 \times 12}$$

δ = helix angle of distortion in radians,

α = radial angle of distortion in radians,

l = length of shaft in inches,

e = radius of shaft in inches,

p_s = greatest shearing stress in pounds per square inch existing in shaft,

E_s = modulus of elasticity in shear,

I_p = polar moment of inertia of circular section (see table of standard sections),

P = force in pounds producing torsion, that is, the turning force,

a = lever arm of force P in inches,

d = diameter of shaft in inches,

N = revolutions per minute.

In deriving the above formulæ, the torsion is treated as due to a couple of the same turning moment, Pa , as the single force P with lever arm a . This eliminates the consideration of any stresses other than shearing stresses, and, in applying these formulæ to the case of a single driving force, bending stresses and bearing friction are neglected.

Flexure of Beams

When a beam is strained by a vertical load, the greatest strain will be in the extreme upper and lower fibers of the beam. The intensity of the strain that can be borne by the extreme fibers is the limit of the strength of the beam. The upper fibers are compressed and the lower fibers are stretched when a beam is loaded between supports; the converse holds when it is loaded beyond supports. Somewhere along or near the center of the beam the fibers are neither extended nor compressed; the plane of these fibers is called the **neutral surface**. The line of intersection of the neutral surface with any cross-section of the beam is the **neutral axis** of the section.

If the stresses remain within the elastic limits of the material in both tension and compression, and provided the modulus of elasticity is the same for both kinds of stress, then the **neutral axis** of the section passes through its **center of gravity**.

The **elastic curve** is the curve assumed by a beam under load.

The **bending moment** for any section of a beam is the algebraic sum of the moments of the external or applied forces acting on the beam on one side of the section. Thus, for the beam shown, the bending moment about *A* is

$$M = R_1x - Pa$$

The bending moment, *M*, of any section is numerically equal to the **moment of resistance** of

the section, which is the resistance which the particles of the beam offer to distortion.

The **moment of resistance** equals

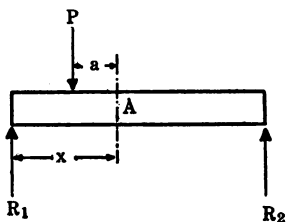
$$\frac{\phi I}{e} = M = \text{bending moment}$$

ϕ = stress per unit area at the outermost element of the section,

e = distance of extreme element of beam from neutral axis,

I = rectangular moment of inertia of beam section about its horizontal gravity axis.

In designing the proper cross-section for a beam, the maximum bending moment (given for standard cases



under Beam Loadings) is equated to $\frac{pI}{e}$. The term $\frac{I}{e}$, called the section modulus, may be obtained from the table of standard sections of beams. The value of p must not exceed the maximum allowable stress per unit area for the material of the beam. The **maximum allowable stress** equals the ultimate strength divided by the factor of safety.

The equation of the elastic curve and its radius of curvature may be found from the relations:

$$M = \frac{pI}{e} = \frac{EI}{\rho} = EI \frac{d^2y}{dx^2} \text{ (approx.)}$$

E = modulus of elasticity of material of beam in tension or compression,

ρ = radius of curvature of the elastic curve,

(x, y) = coördinates of any point on the elastic curve.

The **deflection** of a beam at any point is obtained by substituting, in the equation of the elastic curve, the particular value of x in question, and solving for the corresponding value of y , which equals the deflection. The **maximum deflection** occurs at the section for which $\frac{dy}{dx} = 0$.

Shear

The **vertical shear** in a beam is equal to the first derivative of the bending moment in respect to x , thus

$$\text{Vertical shear} = J = \frac{dM}{dx}$$

where M is the bending moment (expressed as a function of x).

The value of the vertical shear for any particular

section is found by substituting the corresponding value of x in the expression for $\frac{dM}{dx}$. The result is the required vertical shear.

The **maximum bending moment** is found by equating $\frac{dM}{dx} = 0$, and then solving for the corresponding value of x . This particular value of x is substituted in the equation of the bending moment, M , and the resulting expression equals the maximum bending moment.

The **horizontal shear** in a plane parallel to the neutral surface (that is, the surface in which neither tension nor compression occurs), and at a distance z'' from it, equals

$$X \text{ (in pounds/sq. inch)} = \frac{J}{y''I} \int_{z''}^{\circ} z dF$$

where J = total vertical shear in pounds,

y'' = width of beam section at z'' in inches,

I = rectangular moment of inertia of entire section about the horizontal gravity axis,

$\int_{z''}^{\circ} z dF$ = area in square inches of that portion of the section above z'' multiplied by the distance in inches of its center of gravity above the neutral axis.

Beam Loadings

M = bending moment,

M_m = maximum bending moment,

y = deflection at any point,

d = maximum deflection,

P = concentrated load,

W = uniformly distributed load.

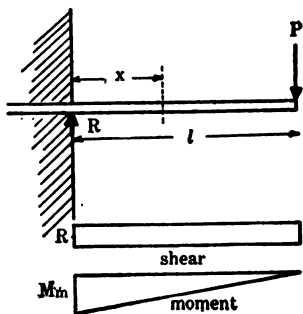
Cantilever Beam with Concentrated Load at the Free End

$$M = P(l - x)$$

$$M_m = Pl$$

$$y = \frac{P}{EI} \left(\frac{lx^2}{2} - \frac{x^3}{6} \right)$$

$$d = \frac{Pl^3}{3EI}$$



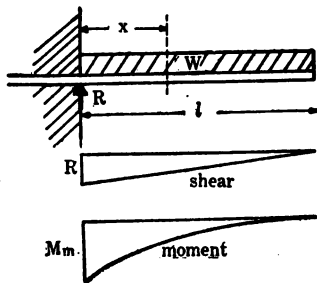
Cantilever Beam with Uniform Load

$$M = \frac{W(l-x)^2}{2l}$$

$$M_m = \frac{Wl}{2}$$

$$y = \frac{Wx^2 [2l^2 + (2l-x)^2]}{24EI}$$

$$d = \frac{Wl^3}{8EI}$$



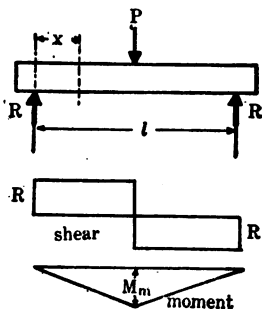
Beam Supported at Both Ends and Loaded with a Concentrated Load at Center

$$M = \frac{P}{2}x$$

$$M_m = \frac{Pl}{4}$$

$$y = \frac{Px(3l^2 - 4x^2)}{48EI}$$

$$d = \frac{Pl^3}{48EI}$$



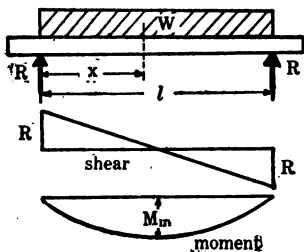
Beam Supported at Both Ends and Uniformly Loaded

$$M = \frac{Wx(l-x)}{2l}$$

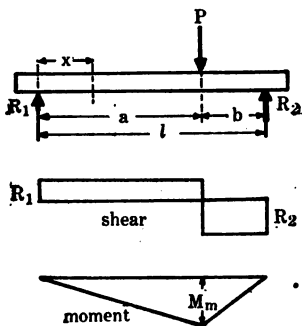
$$M_m = \frac{Wl}{8}$$

$$y = \frac{Wx(l^2 - 2lx^2 + x^3)}{24EI}$$

$$d = \frac{5Wl^3}{384EI}$$



Beam Supported at Both Ends and Loaded at Any Point



$$M = \frac{Pbx}{l} \quad x < a$$

$$M = \frac{Pbx}{l} - P(x - a) \quad x > a$$

$$M_m = \frac{Pab}{l}$$

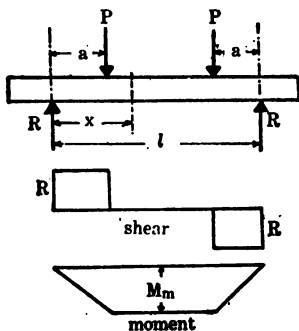
$$y = \frac{Pbx}{6EI} (2la - a^2 - x^2) \quad x < a$$

$$y = \frac{Pa(l-x)}{6EI} (2lx - x^2 - a^2) \quad x > a$$

$$d = \frac{Pb}{27EI} \sqrt{3(2ab + a^2)^3}$$

occurring when $x = \frac{1}{3} \sqrt{3(2ab + a^2)}$

Beam Supported at Both Ends and Loaded with Two Concentrated Loads at Equal Distances from Each End



$$M = Px \quad x < a$$

$$M = Pa \quad x > a$$

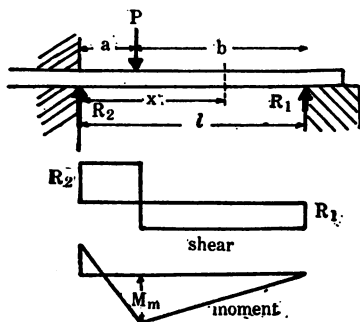
$$M_m = Pa$$

$$y = \frac{Px}{6EI} (3la - 3a^2 - x^2) \quad x < a$$

$$y = \frac{Pa}{6EI} (3lx - 3x^2 - a^2) \quad x > a$$

$$d = \frac{Pa}{6EI} \left(\frac{3}{4} l^2 - a^2 \right)$$

Beam Fixed at One End, Supported at the Other, and with a Concentrated Load at Any Point



$$R_1 = \frac{Pa^2(3l-a)}{2l^3}$$

$$R_2 = P - R_1$$

$$M = P(a-x) - R_1(l-x) \quad x < a$$

$$M = R_1(x-l) \quad x > a$$

$$M_m = R_1(l-a)$$

$$y = \frac{1}{6EI} (R_1x^3 - 3R_1lx^2 + 3Pax^2 - Px^3) \quad x < a$$

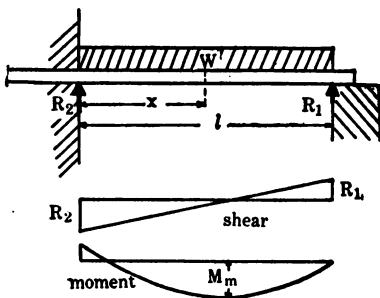
$$y = \frac{1}{6EI} (R_1x^3 - 3R_1lx^2 + 3Pa^2x - Pa^3) \quad x > a$$

$$d = \frac{Pa^2}{6EI} (l-a) \sqrt{\frac{(l-a)}{(3l-a)}}$$

occurring when

$$x = l \left(1 - \sqrt{\frac{l-a}{3l-a}} \right)$$

Beam Fixed at One End, Supported at the Other and Uniformly Loaded



$$R_1 = \frac{3}{8} W$$

$$R_2 = \frac{5}{8} W$$

$$M = \frac{W}{8l} (l - 4x)(l - x)$$

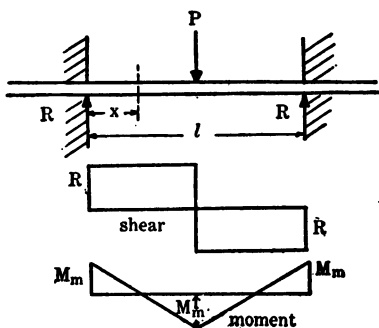
$$M_m = \frac{Wl}{8}$$

$$y = \frac{Wx^2}{48EI} (l - x)(3l - 2x)$$

$$d = 0.0054 \frac{Wl^3}{EI}$$

occurring when $x = 0.5785l$

Beam Fixed at Both Ends and Loaded at the Center

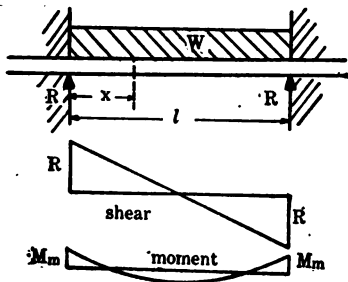


$$M = \frac{P}{8} (4x - l)$$

$$M_m = \frac{Pl}{8}$$

$$y = \frac{Px^2}{48EI} (4x - 3l)$$

$$d = \frac{Pl^3}{192EI}$$

Beam Fixed at Both Ends and Uniformly Loaded

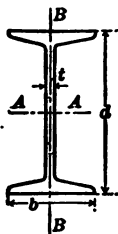
$$M = \frac{W}{12l} (6lx - 6x^2 - l^2)$$

$$M_m = \frac{Wl}{12}$$

$$y = \frac{Wx^2}{24EI} (l - x)^2$$

$$d = \frac{Wl^3}{384EI}$$

PROPERTIES OF STANDARD I BEAMS *



| Depth of beam, <i>d</i> inches | Weight per foot, <i>w</i> pounds | Area of section, <i>A</i> inches ² | Width of flange, <i>b</i> inches | Thickness of web, <i>t</i> inches | Axis A-A | | | Axis B-B | | |
|--------------------------------|----------------------------------|---|----------------------------------|-----------------------------------|---|-------------------------------------|---|---|-------------------------------------|---|
| | | | | | Moment of inertia, <i>I</i> inches ⁴ | Radius of gyration, <i>k</i> inches | Section modulus, <i>s</i> inches ³ | Moment of inertia, <i>I</i> inches ⁴ | Radius of gyration, <i>k</i> inches | Section modulus, <i>s</i> inches ³ |
| 20 | 75.0 | 22.06 | 6.399 | 0.649 | 1268.8 | 7.58 | 126.9 | 30.3 | 1.17 | 9.5 |
| | 70.0 | 20.59 | 6.325 | 0.575 | 1219.8 | 7.70 | 122.0 | 29.0 | 1.19 | 9.2 |
| | 65.0 | 19.08 | 6.250 | 0.500 | 1169.5 | 7.83 | 117.0 | 27.9 | 1.21 | 8.9 |
| 18 | 90.0 | 26.47 | 7.245 | 0.807 | 1260.4 | 6.90 | 140.0 | 52.0 | 1.40 | 14.4 |
| | 85.0 | 25.00 | 7.163 | 0.725 | 1220.7 | 6.99 | 135.6 | 50.0 | 1.42 | 14.0 |
| | 80.0 | 23.53 | 7.082 | 0.644 | 1181.0 | 7.09 | 131.2 | 48.1 | 1.43 | 13.6 |
| | 75.0 | 22.05 | 7.000 | 0.562 | 1141.3 | 7.19 | 126.8 | 46.2 | 1.45 | 13.2 |
| 18 | 70.0 | 20.59 | 6.259 | 0.719 | 921.2 | 6.69 | 102.4 | 24.6 | 1.09 | 7.9 |
| | 65.0 | 19.12 | 6.177 | 0.637 | 881.5 | 6.79 | 97.9 | 23.5 | 1.11 | 7.6 |
| | 60.0 | 17.65 | 6.095 | 0.555 | 841.8 | 6.91 | 93.5 | 22.4 | 1.13 | 7.3 |
| | 55.0 | 15.93 | 6.000 | 0.460 | 795.6 | 7.07 | 88.4 | 21.2 | 1.15 | 7.1 |
| 15 | 75.0 | 22.06 | 6.292 | 0.882 | 691.2 | 5.60 | 92.2 | 30.7 | 1.18 | 9.8 |
| | 70.0 | 20.59 | 6.194 | 0.784 | 663.7 | 5.68 | 88.5 | 29.0 | 1.19 | 9.4 |
| | 65.0 | 19.12 | 6.096 | 0.686 | 636.1 | 5.77 | 84.8 | 27.4 | 1.20 | 9.0 |
| | 60.0 | 17.67 | 6.000 | 0.590 | 609.0 | 5.87 | 81.2 | 26.0 | 1.21 | 8.7 |
| 15 | 55.0 | 16.18 | 5.746 | 0.656 | 511.0 | 5.62 | 68.1 | 17.1 | 1.02 | 5.9 |
| | 50.0 | 14.71 | 5.648 | 0.558 | 483.4 | 5.73 | 64.5 | 16.0 | 1.04 | 5.7 |
| | 45.0 | 13.24 | 5.550 | 0.460 | 455.9 | 5.87 | 60.8 | 15.1 | 1.07 | 5.4 |
| | 42.0 | 12.48 | 5.500 | 0.410 | 441.8 | 5.95 | 58.9 | 14.6 | 1.08 | 5.3 |
| 12 | 55.0 | 16.18 | 5.611 | 0.821 | 321.0 | 4.45 | 53.5 | 17.5 | 1.04 | 6.2 |
| | 50.0 | 14.71 | 5.489 | 0.699 | 303.4 | 4.54 | 50.6 | 16.1 | 1.05 | 5.9 |
| | 45.0 | 13.24 | 5.366 | 0.576 | 285.7 | 4.65 | 47.6 | 14.9 | 1.06 | 5.6 |
| | 40.0 | 11.84 | 5.250 | 0.460 | 269.0 | 4.77 | 44.8 | 13.8 | 1.08 | 5.3 |
| 12 | 35.0 | 10.29 | 5.086 | 0.436 | 228.3 | 4.71 | 38.0 | 10.1 | 0.99 | 4.0 |
| | 31.5 | 9.26 | 5.000 | 0.350 | 215.8 | 4.83 | 36.0 | 9.5 | 1.01 | 3.8 |

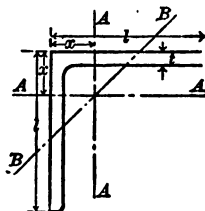
* Manufactured by the Carnegie Steel Company, Pittsburg, Pa.

PROPERTIES OF STANDARD I BEAMS * (Continued)

| Depth of beam, d inches | Weight per foot, w pounds | Area of section, A inches ² | Width of flange, b inches | Thickness of web, t inches | Axis A-A | | | Axis B-B | | |
|---------------------------|-----------------------------|--|-----------------------------|------------------------------|--|--------------------------------|--|--|--------------------------------|--|
| | | | | | Moment of inertia, I inches ⁴ | Radius of gyration, k inches | Section modulus, s inches ³ | Moment of inertia, I inches ⁴ | Radius of gyration, k inches | Section modulus, s inches ³ |
| 10 | 40.0 | 11.76 | 5.099 | 0.749 | 158.7 | 3.67 | 31.7 | 9.5 | 0.90 | 3.7 |
| | 35.0 | 10.29 | 4.952 | 0.602 | 146.4 | 3.77 | 29.3 | 8.5 | 0.91 | 3.4 |
| | 30.0 | 8.82 | 4.805 | 0.455 | 134.2 | 3.90 | 26.8 | 7.7 | 0.93 | 3.2 |
| | 25.0 | 7.37 | 4.660 | 0.310 | 122.1 | 4.07 | 24.4 | 6.9 | 0.97 | 3.0 |
| 9 | 35.0 | 10.29 | 4.772 | 0.732 | 111.8 | 3.29 | 24.8 | 7.3 | 0.84 | 3.1 |
| | 30.0 | 8.82 | 4.609 | 0.569 | 101.9 | 3.40 | 22.6 | 6.4 | 0.85 | 2.8 |
| | 25.0 | 7.35 | 4.446 | 0.406 | 91.9 | 3.54 | 20.4 | 5.7 | 0.88 | 2.5 |
| | 21.0 | 6.31 | 4.330 | 0.290 | 84.9 | 3.67 | 18.9 | 5.2 | 0.90 | 2.4 |
| 8 | 25.5 | 7.50 | 4.271 | 0.541 | 68.4 | 3.02 | 17.1 | 4.8 | 0.80 | 2.2 |
| | 23.0 | 6.76 | 4.179 | 0.449 | 64.5 | 3.09 | 16.1 | 4.4 | 0.81 | 2.1 |
| | 20.5 | 6.03 | 4.087 | 0.357 | 60.6 | 3.17 | 15.2 | 4.1 | 0.82 | 2.0 |
| | 18.0 | 5.33 | 4.000 | 0.270 | 56.9 | 3.27 | 14.2 | 3.8 | 0.84 | 1.9 |
| 7 | 20.0 | 5.88 | 3.868 | 0.458 | 42.2 | 2.68 | 12.1 | 3.2 | 0.74 | 1.7 |
| | 17.5 | 5.15 | 3.763 | 0.353 | 39.2 | 2.76 | 11.2 | 2.9 | 0.76 | 1.6 |
| | 15.0 | 4.42 | 3.660 | 0.250 | 36.2 | 2.86 | 10.4 | 2.7 | 0.78 | 1.5 |
| 6 | 17.25 | 5.07 | 3.575 | 0.475 | 26.2 | 2.27 | 8.7 | 2.4 | 0.68 | 1.3 |
| | 14.75 | 4.34 | 3.452 | 0.352 | 24.0 | 2.35 | 8.0 | 2.1 | 0.69 | 1.2 |
| | 12.25 | 3.61 | 3.330 | 0.230 | 21.8 | 2.46 | 7.3 | 1.9 | 0.72 | 1.1 |
| 5 | 14.75 | 4.34 | 3.294 | 0.504 | 15.2 | 1.87 | 6.1 | 1.7 | 0.63 | 1.0 |
| | 12.25 | 3.60 | 3.147 | 0.357 | 13.6 | 1.94 | 5.5 | 1.5 | 0.63 | 0.92 |
| | 9.75 | 2.87 | 3.000 | 0.210 | 12.1 | 2.05 | 4.8 | 1.2 | 0.65 | 0.82 |

* Manufactured by the Carnegie Steel Company, Pittsburg, Pa.

PROPERTIES OF STANDARD ANGLES WITH EQUAL LEGS *



| Size, t inches | Thickness, t inches | Weight per foot, w pounds | Area of section, A inches ² | Axis A-A | | | | Axis B-B |
|------------------|-----------------------|-----------------------------|--|--|--|--------------------------------|--|--------------------------------------|
| | | | | Distance from back of angle to center of gravity, x ins. | Moment of inertia, I inches ⁴ | Radius of gyration, k inches | Section modulus, s inches ³ | Minimum radius of gyration, k ins. |
| 6×6 | 1 1/4 | 37.4 | 11.00 | 1.86 | 35.5 | 1.80 | 8.6 | 1.16 |
| | 1 1/2 | 35.3 | 10.37 | 1.84 | 33.7 | 1.80 | 8.1 | 1.16 |
| | 1 3/4 | 33.1 | 9.73 | 1.82 | 31.9 | 1.81 | 7.6 | 1.17 |
| | 1 7/8 | 31.0 | 9.09 | 1.80 | 30.1 | 1.82 | 7.2 | 1.17 |
| | 2 | 28.7 | 8.44 | 1.78 | 28.2 | 1.83 | 6.7 | 1.17 |
| | 2 1/8 | 26.5 | 7.78 | 1.75 | 26.2 | 1.83 | 6.2 | 1.17 |
| | 2 1/4 | 24.2 | 7.11 | 1.73 | 24.2 | 1.84 | 5.7 | 1.17 |
| | 2 3/8 | 21.9 | 6.43 | 1.71 | 22.1 | 1.85 | 5.1 | 1.18 |
| | 2 1/2 | 19.6 | 5.75 | 1.68 | 19.9 | 1.86 | 4.6 | 1.18 |
| | 2 7/8 | 17.2 | 5.06 | 1.66 | 17.7 | 1.87 | 4.1 | 1.19 |
| | 14.9 | 4.36 | 1.64 | 15.4 | 1.88 | 3.5 | 1.19 | |
| 4×4 | 1 1/4 | 18.5 | 5.44 | 1.27 | 7.7 | 1.19 | 2.8 | 0.77 |
| | 1 1/2 | 17.1 | 5.03 | 1.25 | 7.2 | 1.19 | 2.6 | 0.77 |
| | 1 3/4 | 15.7 | 4.61 | 1.23 | 6.7 | 1.20 | 2.4 | 0.77 |
| | 1 7/8 | 14.3 | 4.18 | 1.21 | 6.1 | 1.21 | 2.2 | 0.78 |
| | 2 | 12.8 | 3.75 | 1.18 | 5.6 | 1.22 | 2.0 | 0.78 |
| | 2 1/8 | 11.3 | 3.31 | 1.16 | 5.0 | 1.23 | 1.8 | 0.78 |
| | 2 1/4 | 9.8 | 2.86 | 1.14 | 4.4 | 1.23 | 1.5 | 0.79 |
| | 2 3/8 | 8.2 | 2.40 | 1.12 | 3.7 | 1.24 | 1.3 | 0.79 |
| 3 1/2 × 3 1/2 | 1 1/4 | 13.6 | 3.98 | 1.10 | 4.3 | 1.04 | 1.8 | 0.68 |
| | 1 1/2 | 12.4 | 3.62 | 1.08 | 4.0 | 1.05 | 1.6 | 0.68 |
| | 1 3/4 | 11.1 | 3.25 | 1.06 | 3.6 | 1.06 | 1.5 | 0.68 |
| | 1 7/8 | 9.8 | 2.87 | 1.04 | 3.3 | 1.07 | 1.3 | 0.68 |
| | 2 | 8.5 | 2.48 | 1.01 | 2.9 | 1.07 | 1.2 | 0.69 |
| | 2 1/8 | 7.2 | 2.09 | 0.99 | 2.5 | 1.08 | 0.98 | 0.69 |

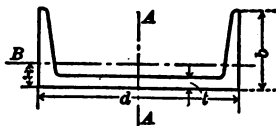
* Manufactured by the Carnegie Steel Company, Pittsburg, Pa.

PROPERTIES OF STANDARD ANGLES WITH EQUAL LEGS * (Continued)

| Size, <i>l</i> inches | Thickness, <i>t</i> inches | Weight per foot, <i>w</i> pounds | Area of section, <i>A</i> inches ² | Axis A-A | | | | Axis B-B Minimum radius of gyration, <i>k</i> ins. |
|-----------------------|----------------------------|----------------------------------|---|---|---|-------------------------------------|---|---|
| | | | | Distance from back of angle to center of gravity, <i>x</i> ins. | Moment of inertia, <i>I</i> inches ⁴ | Radius of gyration, <i>k</i> inches | Section modulus, <i>S</i> inches ³ | |
| 3 × 3 | $\frac{3}{8}$ | 9.4 | 2.75 | 0.93 | 2.2 | 0.90 | 1.1 | 0.58 |
| | $\frac{7}{16}$ | 8.3 | 2.43 | 0.91 | 2.0 | 0.91 | 0.95 | 0.58 |
| | $\frac{1}{4}$ | 7.2 | 2.11 | 0.89 | 1.8 | 0.91 | 0.83 | 0.58 |
| | $\frac{5}{16}$ | 6.1 | 1.78 | 0.87 | 1.5 | 0.92 | 0.71 | 0.59 |
| | $\frac{3}{8}$ | 4.9 | 1.44 | 0.84 | 1.2 | 0.93 | 0.58 | 0.59 |
| 2½ × 2½ | $\frac{3}{8}$ | 5.9 | 1.73 | 0.76 | 0.98 | 0.75 | 0.57 | 0.48 |
| | $\frac{7}{16}$ | 5.0 | 1.47 | 0.74 | 0.85 | 0.76 | 0.48 | 0.49 |
| | $\frac{1}{4}$ | 4.1 | 1.19 | 0.72 | 0.70 | 0.77 | 0.39 | 0.49 |
| | $\frac{5}{16}$ | 3.07 | 0.90 | 0.69 | 0.55 | 0.78 | 0.30 | 0.49 |
| 2 × 2 | $\frac{3}{8}$ | 4.7 | 1.36 | 0.64 | 0.48 | 0.59 | 0.35 | 0.39 |
| | $\frac{7}{16}$ | 3.92 | 1.15 | 0.61 | 0.42 | 0.60 | 0.30 | 0.39 |
| | $\frac{1}{4}$ | 3.19 | 0.94 | 0.59 | 0.35 | 0.61 | 0.25 | 0.39 |
| | $\frac{5}{16}$ | 2.44 | 0.71 | 0.57 | 0.28 | 0.62 | 0.19 | 0.40 |

* Manufactured by the Carnegie Steel Company, Pittsburg, Pa.

PROPERTIES OF STANDARD CHANNELS *



| Depth of channel, <i>d</i> inches | Weight per foot, <i>w</i> pounds | Area of section, <i>A</i> inches ² | Width of flange, <i>b</i> inches | Thickness of web, <i>t</i> inches | Axis A-A | | | Axis B-B | | | |
|--------------------------------------|-------------------------------------|--|-------------------------------------|--------------------------------------|--|--|--|---|--|--|--|
| | | | | | Moment of inertia, <i>I</i> inches ⁴ | Radius of gyration, <i>k</i> inches | Section modulus, <i>s</i> inches ³ | Distance from back of web to center of gravity, <i>x</i> inches | Moment of inertia, <i>I</i> inches ⁴ | Radius of gyration, <i>k</i> inches | Section modulus, <i>s</i> inches ³ |
| 12 | 40 | 11.76 | 3.418 | 0.758 | 196.9 | 4.09 | 32.8 | 0.72 | 6.6 | 0.75 | 2.5 |
| | 35 | 10.29 | 3.296 | 0.636 | 179.3 | 4.17 | 29.9 | 0.69 | 5.9 | 0.76 | 2.3 |
| | 30 | 8.82 | 3.173 | 0.513 | 161.7 | 4.28 | 26.9 | 0.68 | 5.2 | 0.77 | 2.1 |
| | 25 | 7.35 | 3.050 | 0.390 | 144.0 | 4.43 | 24.0 | 0.68 | 4.5 | 0.79 | 1.9 |
| | 20.5 | 6.03 | 2.940 | 0.280 | 128.1 | 4.61 | 21.4 | 0.70 | 3.9 | 0.81 | 1.7 |
| 10 | 35 | 10.29 | 3.183 | 0.823 | 115.5 | 3.35 | 23.1 | 0.70 | 4.7 | 0.67 | 1.9 |
| | 30 | 8.82 | 3.036 | 0.676 | 103.2 | 3.42 | 20.7 | 0.65 | 4.0 | 0.67 | 1.7 |
| | 25 | 7.35 | 2.889 | 0.529 | 91.0 | 3.52 | 18.2 | 0.62 | 3.4 | 0.68 | 1.5 |
| | 20 | 5.88 | 2.742 | 0.382 | 78.7 | 3.66 | 15.7 | 0.61 | 2.9 | 0.70 | 1.3 |
| | 15 | 4.46 | 2.600 | 0.240 | 66.9 | 3.87 | 13.4 | 0.64 | 2.3 | 0.72 | 1.2 |
| 9 | 25 | 7.35 | 2.815 | 0.615 | 70.7 | 3.10 | 15.7 | 0.62 | 3.0 | 0.64 | 1.4 |
| | 20 | 5.88 | 2.652 | 0.452 | 60.8 | 3.21 | 13.5 | 0.59 | 2.5 | 0.65 | 1.2 |
| | 15 | 4.41 | 2.488 | 0.288 | 50.9 | 3.40 | 11.3 | 0.59 | 2.0 | 0.67 | 1.0 |
| | 13.25 | 3.89 | 2.430 | 0.230 | 47.3 | 3.49 | 10.5 | 0.61 | 1.8 | 0.67 | 0.97 |
| 8 | 21.25 | 6.25 | 2.622 | 0.582 | 47.8 | 2.77 | 11.9 | 0.59 | 2.3 | 0.60 | 1.1 |
| | 18.75 | 5.51 | 2.530 | 0.490 | 43.8 | 2.82 | 11.0 | 0.57 | 2.0 | 0.60 | 1.0 |
| | 16.25 | 4.78 | 2.439 | 0.399 | 39.9 | 2.89 | 10.0 | 0.56 | 1.8 | 0.61 | 0.95 |
| | 13.75 | 4.04 | 2.347 | 0.307 | 36.0 | 2.98 | 9.0 | 0.56 | 1.6 | 0.62 | 0.87 |
| | 11.25 | 3.35 | 2.260 | 0.220 | 32.3 | 3.11 | 8.1 | 0.58 | 1.3 | 0.63 | 0.79 |
| 7 | 19.75 | 5.81 | 2.513 | 0.633 | 33.2 | 2.39 | 9.5 | 0.58 | 1.9 | 0.56 | 0.96 |
| | 17.25 | 5.07 | 2.408 | 0.528 | 30.2 | 2.44 | 8.6 | 0.56 | 1.6 | 0.57 | 0.87 |
| | 14.75 | 4.34 | 2.303 | 0.423 | 27.2 | 2.50 | 7.8 | 0.54 | 1.4 | 0.57 | 0.79 |
| | 12.25 | 3.60 | 2.198 | 0.318 | 24.2 | 2.59 | 6.9 | 0.53 | 1.2 | 0.58 | 0.71 |
| | 9.75 | 2.85 | 2.090 | 0.210 | 21.1 | 2.72 | 6.0 | 0.55 | 0.98 | 0.59 | 0.63 |
| 6 | 15.5 | 4.56 | 2.283 | 0.563 | 19.5 | 2.07 | 6.5 | 0.55 | 1.3 | 0.53 | 0.74 |
| | 13.0 | 3.82 | 2.160 | 0.440 | 17.3 | 2.13 | 5.8 | 0.52 | 1.1 | 0.53 | 0.65 |
| | 10.5 | 3.09 | 2.038 | 0.318 | 15.1 | 2.21 | 5.0 | 0.50 | 0.88 | 0.53 | 0.57 |
| | 8.0 | 2.38 | 1.920 | 0.200 | 13.0 | 2.34 | 4.3 | 0.52 | 0.70 | 0.54 | 0.50 |
| 5 | 11.5 | 3.38 | 2.037 | 0.477 | 10.4 | 1.75 | 4.2 | 0.51 | 0.82 | 0.49 | 0.54 |
| | 9.0 | 2.65 | 1.890 | 0.330 | 8.9 | 1.83 | 3.6 | 0.48 | 0.64 | 0.49 | 0.45 |
| | 6.5 | 1.95 | 1.750 | 0.190 | 7.4 | 1.95 | 3.0 | 0.49 | 0.48 | 0.50 | 0.38 |
| 4 | 7.25 | 2.13 | 1.725 | 0.325 | 4.6 | 1.46 | 2.3 | 0.46 | 0.44 | 0.46 | 0.35 |
| | 6.25 | 1.84 | 1.652 | 0.252 | 4.2 | 1.51 | 2.1 | 0.46 | 0.38 | 0.45 | 0.32 |
| | 5.25 | 1.55 | 1.580 | 0.180 | 3.8 | 1.56 | 1.9 | 0.46 | 0.32 | 0.45 | 0.29 |

* Manufactured by the Carnegie Steel Company, Pittsburgh, Pa.

COLUMNS

Note. The breaking load in Euler's and in Gordon's formula, and the safe load in Ritter's formula are in pounds. In all of the formulæ for columns, the length, l , and radius of gyration, k , must be expressed in the same units (generally inches).

Euler's Formula

(1) Column with round ends,

$$\text{breaking load} = EI \frac{\pi^2}{l^2} = \pi^2 EF \left(\frac{k^2}{l^2} \right)$$

(2) Column with flat ends,

$$\text{breaking load} = 4 EI \frac{\pi^2}{l^2} = 4 \pi^2 EF \left(\frac{k^2}{l^2} \right).$$

(3) Pin-and-square column (column with one end round and the other flat),

$$\text{breaking load} = \frac{9}{4} EI \frac{\pi^2}{l^2} = \frac{9}{4} \pi^2 EF \left(\frac{k^2}{l^2} \right)$$

in which

E = modulus of elasticity of material of column in tension or compression,

I = rectangular moment of inertia of cross-section about neutral axis,

l = length of column,

F = area of cross-section in sq. inches,

k = least radius of gyration of section.

Gordon's or Rankine's Formula

(1) Column with flat ends,

$$\text{breaking load} = \frac{FC}{1 + \beta \left(\frac{l}{k}\right)^2}$$

(2) Column with rounded ends,

$$\text{breaking load} = \frac{FC}{1 + 4\beta \left(\frac{l}{k}\right)^2}$$

(3) Pin-and-square column,

$$\text{breaking load} = \frac{FC}{1 + 1.78\beta \left(\frac{l}{k}\right)^2}$$

in which

F = area of cross-section in square inches,

C = ultimate compressive strength of material of column in pounds per square inch,

l = length of column,

k = least radius of gyration of section,

β = empirical constant.

Values of β and of C , in Gordon's formula, are as follows for different materials:

| Material { | Hard steel | Medium steel | Soft steel | Wrought iron | Cast iron | Timber |
|--------------------|--------------------|--------------------|--------------------|--------------------|------------------|------------------|
| C (lbs./sq. in.) | 70,000 | 50,000 | 45,000 | 36,000 | 70,000 | 7200 |
| β | $\frac{1}{25,000}$ | $\frac{1}{36,000}$ | $\frac{1}{36,000}$ | $\frac{1}{36,000}$ | $\frac{1}{6400}$ | $\frac{1}{3000}$ |

Ritter's Formula

(1) Column with flat ends,

$$\text{safe load} = \frac{FC}{1 + \frac{C'}{4\pi^2 E} \left(\frac{l}{k}\right)^2}$$

(2) Column with rounded ends,

$$\text{safe load} = \frac{FC}{1 + \frac{C'}{\pi^2 E} \left(\frac{l}{k}\right)^2}$$

(3) Pin-and-square column,

$$\text{safe load} = \frac{FC}{1 + \frac{1.78 C'}{4\pi^2 E} \left(\frac{l}{k}\right)^2}$$

in which

 F = area of cross-section in square inches, C = maximum safe compressive stress of material of column in pounds per square inch, C' = compressive stress at elastic limit in pounds per square inch, E = modulus of elasticity for tension or compression, l = length of column, k = least radius of gyration.**J. B. Johnson's Formula**

Breaking load in pounds; cross-section in square inches.

For **mild steel**:

(1) Pin-ends,

$$\text{breaking load} = \left[42,000 - 0.97 \left(\frac{l}{k}\right)^2 \right] F, \quad \left(\frac{l}{k}\right) \text{ not } > 150$$

(2) Flat ends,

$$\text{breaking load} = \left[42,000 - 0.62 \left(\frac{l}{k} \right)^2 \right] F$$

$$\left(\frac{l}{k} \right) \text{ not } > 190$$

For wrought iron:

(1) Pin-ends,

$$\text{breaking load} = \left[34,000 - 0.67 \left(\frac{l}{k} \right)^2 \right] F$$

$$\left(\frac{l}{k} \right) \text{ not } > 170$$

(2) Flat ends,

$$\text{breaking load} = \left[34,000 - 0.43 \left(\frac{l}{k} \right)^2 \right] F$$

$$\left(\frac{l}{k} \right) \text{ not } > 210$$

Notation same as in Ritter's formula.

Straight-line Formula

Breaking load in pounds; cross-section in square inches.

For mild steel:

(1) Hinged ends,

$$\text{breaking load} = \left[52,000 - 220 \left(\frac{l}{k} \right) \right] F$$

(2) Flat ends,

$$\text{breaking load} = \left[52,000 - 179 \left(\frac{l}{k} \right) \right] F$$

For wrought iron:

(1) Hinged ends,

$$\text{breaking load} = \left[42,000 - 157 \left(\frac{l}{k} \right) \right] F$$

(2) Flat ends,

$$\text{breaking load} = \left[42,000 - 128 \left(\frac{l}{k} \right) \right] F$$

Notation same as in Ritter's formula.

Wooden Columns

The breaking load in pounds for solid wooden columns with square ends is

$$P = \frac{(700 + 15 m) FC}{700 + 15 m + m^2}$$

F = cross-section in square inches,

m = ratio of the length, l , of the column to the least dimension d , of the cross-section (that is, $m = \frac{l}{d}$),

C = ultimate compressive strength of material of column in pounds per square inch.

Values of C , the ultimate compressive strength, for different kinds of timber are as follows:

| | |
|--|------------------|
| White oak and Georgia yellow pine..... | 5000 lb./sq. in. |
| Douglas fir and short-leaf yellow pine... | 4500 lb./sq. in. |
| Red pine, spruce, hemlock, cypress, chestnut, California redwood, and California spruce..... | 4000 lb./sq. in. |
| White pine and cedar..... | 3500 lb./sq. in. |

The proper factor of safety for yellow pine varies from 3.5 to 5, according to the amount of moisture present in the timber, being greater for larger amounts of moisture. For all other timbers, the proper factor of safety varies from 4 to 5.

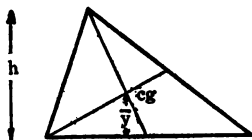
CENTERS OF GRAVITY

Plane Figures

Triangle

The C.G. is on a median line of the triangle, two-thirds of its length from the vertex,

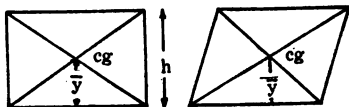
$$\bar{y} = \frac{h}{3}$$



Parallelogram

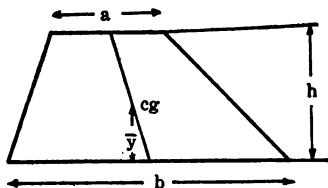
The C.G. is at the intersection of the diagonals,

$$\bar{y} = \frac{h}{2}$$



Trapezoid

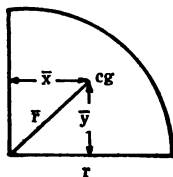
$$\bar{y} = \frac{h(2a+b)}{3(a+b)}$$



Quadrant of Circle

$$\bar{x} = \frac{4r}{3\pi} = \bar{y}$$

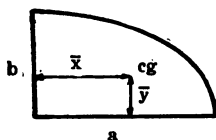
$$\bar{r} = \frac{4r\sqrt{2}}{3\pi}$$



Quadrant of Ellipse

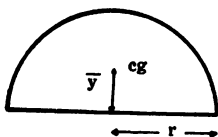
$$\bar{x} = \frac{4a}{3\pi}$$

$$\bar{y} = \frac{4b}{3\pi}$$

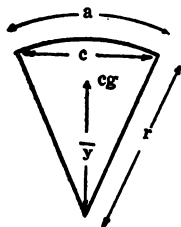


Semicircle

$$\bar{y} = \frac{4r}{3\pi}$$

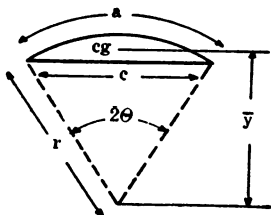
**Circular Sector**

$$\bar{y} = \frac{2rc}{3a}$$

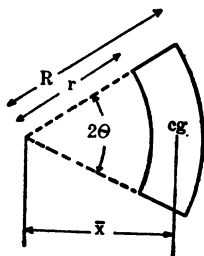
**Circular Segment**

$$\bar{y} = \frac{4}{3} \frac{r \sin^2 \theta}{2\theta - \sin(2\theta)}$$

θ is in radians

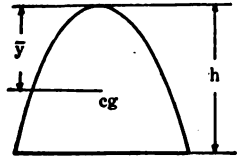
**Sector of a Circular Ring**

$$\bar{x} = \frac{2R^3 - r^3 \sin \theta}{3R^2 - r^2 \theta}$$



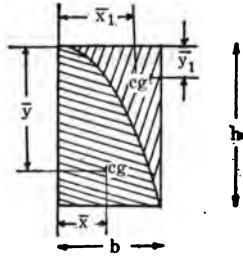
Parabolic Segment

$$\bar{y} = \frac{3h}{5}$$



Parabolic Segment

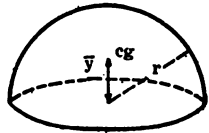
$$\begin{aligned} \bar{x} &= \frac{3}{8}b \\ \bar{y} &= \frac{4}{5}h \\ \bar{x}_1 &= \frac{3}{4}b \\ \bar{y}_1 &= \frac{4}{15}h \end{aligned}$$



Solids

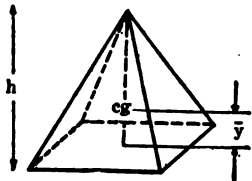
Hemisphere

$$\bar{y} = \frac{3r}{8}$$










Right Pyramid or Cone

$$\bar{y} = \frac{h}{4}$$














MOMENT OF INERTIA OF SOLIDS

$$M = \text{mass of body} = \frac{W}{g}$$



| Shape of figure | Description | Axis of rotation | Moment of inertia |
|---|---|--|---|
|  | uniform thin rod | (1) through center perpendicular to length (2) through end perpendicular to length | $M \frac{l^2}{12}$ $M \frac{l^2}{3}$ |
|  | thin rectangular plate | (1) through center of gravity perpendicular to plate (2) through center of gravity parallel to side b | $M \frac{a^2+b^2}{12}$ $M \frac{a^2}{12}$ |
|  | thin circular plate | (1) through center perpendicular to plane (2) any diameter | $M \frac{r^2}{2}$ $M \frac{r^2}{4}$ |
|  | solid cylinder, radius, r | (1) axis of cylinder (2) through center of gravity, perpendicular to axis of cylinder | $M \frac{r^2}{2}$ $M \left(\frac{l^2}{12} + \frac{r^2}{4} \right)$ |
|  | hollow cylinder, R = outer radius, r = inner radius | (1) axis of cylinder (2) through center of gravity perpendicular to axis | $M \left(\frac{R^2+r^2}{2} \right)$ $M \left(\frac{l^2}{12} + \frac{R^2+r^2}{4} \right)$ |
|  | solid sphere, r = radius | through center | $M \frac{2r^2}{5}$ |
|  | hollow sphere, R = external radius, r = internal radius | through center | $M \frac{2}{5} \left(\frac{R^5-r^5}{R^3-r^3} \right)$ |

PROPERTIES OF STANDARD SECTIONS

| Shape of section | Area of section | Rectangular moment of inertia about horizontal gravity axis I | Square of radius of gyration $k^2 = \frac{I}{A}$ | Section modulus $s = \frac{I}{c}$ | Polar moment of inertia, I_p about center of gravity |
|---|---------------------|---|--|-----------------------------------|---|
|  | b^2 | $\frac{bh^3}{12}$ | $\frac{b^2}{12}$ | $\frac{bh^2}{6}$ | $\frac{bh^3}{12}$ |
|  | bh | $\frac{bh^3}{12}$ | $\frac{h^2}{12}$ | $\frac{bh^2}{6}$ | $\frac{bh(b^2+h^2)}{12}$ |
|  | $b^2 - b_1^2$ | $\frac{b^4 - b_1^4}{12}$ | $\frac{b^2 + b_1^2}{12}$ | $\frac{b^4 - b_1^4}{6b}$ | $\frac{b^4 - b_1^4}{12}$ |
|  | $bh - b_1h_1$ | $\frac{bh^3 - b_1h_1^3}{12}$ | $\frac{bh^2 - b_1h_1^2}{12(bh - b_1h_1)}$ | $\frac{bh^2 - b_1h_1^2}{6h}$ | $\frac{bh(h^2 + b^2)}{12} - \frac{b_1h_1(h_1^2 + b_1^2)}{12}$ |
|  | $\frac{\pi d^2}{4}$ | $\frac{\pi d^4}{64}$ | $\frac{d^2}{16}$ | $\frac{\pi d^3}{32}$ | $\frac{\pi d^4}{32}$ |

| | | | | | |
|---|---------------------------------|----------------------------------|---|------------------------------------|----------------------------------|
|  | $\frac{\pi (d_2^3 - d_1^3)}{4}$ | $\frac{\pi (d_2^4 - d_1^4)}{64}$ | $\frac{d_2^2 + d_1^2}{16}$ | $\frac{\pi (d_2^4 - d_1^4)}{32 d}$ | $\frac{\pi (d_2^4 - d_1^4)}{32}$ |
|  | $\frac{bh}{2}$ | $\frac{bh^3}{36}$ | $\frac{h^2}{18}$ | $\frac{bh^2}{24}$ | |
|  | $0.866 d^2$ | $0.06 d^4$ | $0.0697 d^2$ | $0.12 d^3$ | |
|  | $0.828 d^2$ | $0.055 d^4$ | $0.066 d^2$ | $0.109 d^3$ | |
|  | πab | $\frac{\pi ba^3}{4}$ | $\frac{a^2}{4}$ | $\frac{ba^2}{\pi 4}$ | $\frac{\pi}{4} (a^3 b + b^3 a)$ |
|  | $bh - b_1 h_1$ | $\frac{bh^2 - b_1 h_1^2}{12}$ | $\frac{1}{12} \left(\frac{bh^3 - b_1 h_1^3}{bh - b_1 h_1} \right)$ | $\frac{bh^2 - b_1 h_1^2}{6 h}$ | |

PROPERTIES OF STANDARD SECTIONS — Continued

| | | | | |
|---|---------------|------------------------------|---|------------------------------|
|  | $bh - b_1h_1$ | $\frac{bh^3 - b_1h_1^3}{12}$ | $\frac{1}{12} \left(\frac{bh^3 - b_1h_1^3}{bh - b_1h_1} \right)$ | $\frac{bh^3 - b_1h_1^3}{6h}$ |
|  | $b_1h + bh_1$ | $\frac{b_1h^3 + bh_1^3}{12}$ | $\frac{1}{12} \left(\frac{b_1h^3 + bh_1^3}{b_1h + bh_1} \right)$ | $\frac{b_1h^3 + bh_1^3}{6h}$ |

The moment of inertia of such sections as T-beams and angle-bars, the center of gravity of which cannot be determined by inspection, may be obtained as follows: First, find the position of the horizontal gravity axis by the method given on page 74, for Composite Sections. Then divide the section into its component rectangles, with their bases along the gravity axis. The moment of inertia of each rectangle about its base is calculated by the formula $I = \frac{bh^3}{3}$ where b is the base of the rectangle and h its altitude.

The total moment of inertia of the section about its gravity axis is the sum of the moments of inertia of the component rectangles. If there is a rectangular space in the figure, the corresponding moment of inertia is subtracted from that of the solid section.

HYDRAULICS

Head and Pressure

The difference in level of water between two points is called the **head**.

The **pressure** in pounds per square inch at any depth is

$$p = 0.433 h$$

in which

h = head or depth in feet of water,
 0.433 = weight of a column of water 1 foot high and
 1 inch in cross-section.

The pressure on a **submerged surface** is always normal to the surface, and equals

$$P \text{ (in pounds)} = 0.433 hF$$

h = depth of water in feet from the surface of the liquid to the center of gravity of the submerged surface,

F = area of submerged surface in square inches.

Center of Pressure

The **center of pressure** of a submerged surface is the point of application of the resultant of all the fluid pressures on such surface.

The distance of the center of pressure of a **vertical submerged plate** below the liquid surface is

$$d \text{ (in feet)} = \frac{I_s}{F\bar{x}}$$

F = area of plate in square feet,

\bar{z} = distance in feet from the liquid surface to the center of gravity of the plate,

I_o = rectangular moment of inertia of plate about the line of intersection of its plane with the surface of the liquid.

The distance of the center of pressure of a **submerged plate inclined** at an angle θ with the surface is

$$d \text{ (in feet)} = \frac{I_o \sin^2 \theta}{F\bar{z}} + \bar{z}$$

\bar{z} = distance from the liquid surface to the center of gravity of the plate in feet,

F = area of plate in square feet,

I_o = moment of inertia of plate about its gravity axis parallel to the liquid surface.

Flow through Apertures

Due to friction, the velocity of discharge through an aperture in a thin plate or plank is reduced about 3 per cent below its theoretical value. Further, on leaving the orifice, the jet contracts to approximately 64 per cent of the area of the aperture.

The **theoretical** velocity of discharge through a small aperture, in feet per second, is

$$v = \sqrt{2gh}$$

g = acceleration of gravity = 32.16,

h = head in feet.

The **actual** velocity of discharge in feet per second is

$$v = \phi \sqrt{2gh} = 0.97 \sqrt{2gh}$$

ϕ = coefficient of velocity.

The **discharge** through the aperture in cubic feet per second is

$$Q = CF\phi \sqrt{2gh} = 0.62 F \sqrt{2gh}$$

$C = 0.64$ (approx.) = coefficient of contraction,
 F = area of aperture in square feet.

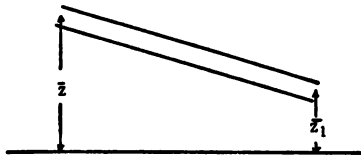
FLOW OF WATER IN PIPES

Bernoulli's Theorem

A general method for calculating the flow of water in pipes is given by Bernoulli's theorem:

$$\frac{v^2}{2g} + \frac{p}{\gamma} + \bar{z} = \frac{v_1^2}{2g} + \frac{p_1}{\gamma} + \bar{z}_1 + k$$

that is, the sum of the velocity head $\frac{v^2}{2g}$, the pressure head $\frac{p}{\gamma}$ and the potential head \bar{z} at any given section of flow is equal to the sum of the corresponding heads at any other section, plus the various losses between the two sections considered.



v = velocity in feet per second at first section,

v_1 = velocity at second section,

p = pressure in pounds per square inch at first section,

p_1 = pressure at second section,

- \bar{z} = potential head at first section in feet, that is,
the distance of the center of the section
above a chosen horizontal reference plane,
 \bar{z}_1 = potential head at second section,
 $g = 32.16$ (approx.),
 γ = weight in pounds of a column of water 1 foot
high and 1 square inch in cross-section =
0.433,
 k = various losses in feet of head between the two
sections of pipe considered.

Losses in Pipes

The following formulæ for losses in pipes enable us to find the value of the term k appearing in Bernoulli's theorem. If several losses occur in a section of pipe, the total loss, k , is the sum of the separate losses.

Loss Due to Friction

The loss of head in feet due to friction in a section of pipe is

$$4f \frac{l}{d} \frac{v^2}{2g}$$

where

- d = diameter of pipe in feet,
 l = length of pipe in feet,
 v = velocity in feet per second,
 f = coefficient of friction, depending on the velocity,
and on the size of pipe.

Values of f , the coefficient of friction, for water in clean iron pipes are as follows (condensed from I. P. Church's "Mechanics of Engineering"):

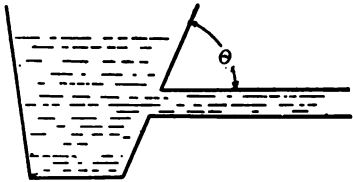
| Velocity in feet per second | Diam. = ½ in. = 0.0417 ft. | Diam. = 1 in. = 0.0834 ft. | Diam. = 2 in. = 0.1667 ft. | Diam. = 4 in. = 0.333 ft. | Diam. = 8 in. = 0.667 ft. | Diam. = 12 in. = 1.00 ft. | Diam. = 16 in. = 1.333 ft. | Diam. = 20 in. = 1.667 ft. |
|-----------------------------|----------------------------|----------------------------|----------------------------|---------------------------|---------------------------|---------------------------|----------------------------|----------------------------|
| 0.1 | 0.0150 | 0.0119 | 0.00870 | 0.00763 | 0.00704 | 0.00669 | 0.00623 | |
| 0.3 | 0.0137 | 0.0113 | 0.00850 | 0.00750 | 0.00693 | 0.00657 | 0.00614 | 0.00578 |
| 0.6 | 0.0124 | 0.0104 | 0.00822 | 0.00732 | 0.00677 | 0.00642 | 0.00603 | 0.00567 |
| 1.0 | 0.0110 | 0.00950 | 0.00790 | 0.00712 | 0.00659 | 0.00624 | 0.00588 | 0.00555 |
| 2.0 | 0.00862 | 0.00810 | 0.00731 | 0.00678 | 0.00624 | 0.00593 | 0.00559 | 0.00529 |
| 3.0 | 0.00753 | 0.00734 | 0.00692 | 0.00650 | 0.00600 | 0.00570 | 0.00538 | 0.00509 |
| 6.0 | 0.00689 | 0.00670 | 0.00640 | 0.00605 | 0.00562 | 0.00534 | 0.00507 | 0.00482 |
| 12.0 | 0.00630 | 0.00614 | 0.00590 | 0.00560 | 0.00522 | 0.00500 | 0.00478 | 0.00457 |
| 20.0 | 0.00615 | 0.00598 | 0.00579 | 0.00549 | 0.00508 | 0.00485 | | |

Loss at Entrance

The loss of head in feet due to entrance from a reservoir into a pipe is equal to

$$\left(\frac{1}{\phi^2} - 1\right) \frac{v^2}{2g} = L_e \frac{v^2}{2g}$$

in which ϕ is the coefficient of friction and is dependent on the angle θ° which the pipe makes with the inner surface of the reservoir.



Values of $L_e \left(= \frac{1}{\phi^2} - 1 \right)$ in the above formula are as follows for different values of θ° (from Church):

| θ° | 90° | 80° | 70° | 60° | 50° | 40° | 30° |
|----------------|-------|-------|-------|-------|-------|-------|-------|
| L_e | 0.505 | 0.565 | 0.635 | 0.713 | 0.794 | 0.870 | 0.987 |

Thus, when the discharge is through a pipe normal to the inner surface of the reservoir, then θ° equals 90° and L_e is, therefore, 0.505, the loss at entrance then being

$$0.505 \frac{v^2}{2g}$$

where v = velocity of flow in pipe in feet per second.

Loss Due to Sudden Enlargement

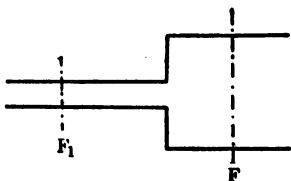
The loss of head in feet due to the sudden enlargement of a pipe is

$$\left(\frac{F}{F_1} - 1\right)^2 \frac{v^2}{2g}$$

F_1 = cross-section area of the smaller pipe in square feet,

F = area of enlarged section in square feet,

v = velocity in feet per second in the enlarged section.



Loss Due to Sudden Contraction

The loss of head in feet due to the sudden contraction of a pipe is

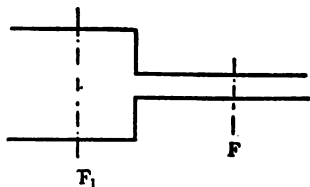
$$\left(\frac{1}{C} - 1\right)^2 \frac{v^2}{2g}$$

in which

v = velocity in feet per second in contracted section,

C = coefficient of contraction, the value of which depends on the

ratio, $\frac{F}{F_1}$, of the small section to the large section.



Values of C , the coefficient of contraction, for

different values of $\frac{F}{F_1}$ are given in the following table (from Church):

| | | | | | | | | | | |
|-----------------|-------|-------|-------|-------|-------|-------|-------|-------|-------|-----|
| $\frac{F}{F_1}$ | 0.10 | 0.20 | 0.30 | 0.40 | 0.50 | 0.60 | 0.70 | 0.80 | 0.90 | 1.0 |
| <i>C</i> | 0.624 | 0.632 | 0.643 | 0.659 | 0.681 | 0.712 | 0.755 | 0.813 | 0.892 | 1.0 |

Loss Due to Bends

The loss of head in feet due to a bend in a circular pipe is

$$\left[0.131 + 1.847 \left(\frac{a}{r} \right)^{\frac{7}{2}} \right] \frac{v^2}{2g} = L_b \frac{v^2}{2g}$$

a = radius of pipe in feet,

r = radius of bend in feet,

v = velocity of flow in feet per second.

Values of L_b for different values of $\frac{a}{r}$ are as follows:

| | | | | | | | | | | |
|---------------|-------|-------|-------|-------|-------|-------|-------|-------|------|------|
| $\frac{a}{r}$ | 0.10 | 0.20 | 0.30 | 0.40 | 0.50 | 0.60 | 0.70 | 0.80 | 0.90 | 1.00 |
| L_b | 0.131 | 0.138 | 0.158 | 0.206 | 0.294 | 0.440 | 0.661 | 0.977 | 1.40 | 1.98 |

Flow Through Straight Cylindrical Pipes

Q = discharge in cubic feet per second,

v = velocity of discharge in feet per second,

l = length of pipe in feet,

d = diameter of pipe in feet,

L_e = coefficient of loss at entrance. In general, the pipe is normal to the inner surface of the reservoir and then $L_e = 0.505$. For other cases see Loss at Entrance.

f = coefficient of friction, obtained from the table on page 145.

(1) Required the head in feet necessary to keep up a given flow of Q cubic feet per second in a clean iron pipe of given length l and diameter d .

The required head is

$$h \text{ (in feet)} = \frac{v^2}{2g} \left(1 + L_e + 4f \frac{l}{d} \right)$$

in which $v = \frac{4Q}{\pi d^2}$

(2) Required the velocity in the pipe, having given the head h and the length l and the diameter d of the pipe; also required the discharge Q in cubic feet per second.

The velocity in feet per second is:

$$v = \sqrt{\frac{2gh}{1 + L_e + 4f \frac{l}{d}}}$$

and after solving for v ,

$$Q = \frac{1}{4} \pi d^2 v$$

Since the value of f depends on the unknown v as well as the known d , we may first put $f = 0.006$ for a trial approximation and solve for v ; then take the value of f corresponding to this velocity and substitute again in the given formula for v . One trial is generally sufficient for ordinary accuracy.

(3) Required the proper diameter d for the pipe to discharge a given quantity Q cubic feet per second, having given the length of pipe and the head h .

The proper diameter in feet is

$$d = \sqrt[5]{\frac{(1 + L_e)d + 4fl}{2gh} \left(\frac{4Q}{\pi} \right)^2}$$

and d being solved for,

$$v = \frac{4Q}{\pi d^2}$$

Since the radical contains d , we must first assume a trial value for d , and taking $f = 0.006$, substitute in the above formula for the diameter. Having obtained a value for d , we solve for the velocity v . With the approximate values of d and v thus obtained, we find the corresponding new value of f from the table of friction, and then substitute again in the formulæ. One or two trials generally give sufficient accuracy.

Flow Through Very Long Pipes

When a pipe is very long (1000 feet or more), the head, velocity, or discharge, etc., may be calculated from the formulæ:

$$h = 4f \frac{l}{d} \frac{v^2}{2g} \quad (\text{Chézy's formula})$$

$$v = \frac{4Q}{\pi d^2}$$

$$h = \frac{64}{\pi^2} f \frac{lQ^2}{d^5 2g}$$

Notation same as in preceding section.

FLOW THROUGH OPEN CHANNELS

Bazin's Formula

The velocity of flow in a channel in feet per second is

$$v = \frac{87 \sqrt{rs}}{0.552 + \frac{m}{\sqrt{r}}}$$

- r = mean hydraulic radius in feet, which is found by dividing the area of the fluid cross-section in square feet by the wetted perimeter in feet (that is, the perimeter of the channel section in contact with the water),
- s = slope of stream (that is, the difference in elevation between two points of the water surface divided by the distance between the two points measured along the surface),
- m = coefficient of roughness, the values of which are given in the following table.

| Character of channel | Value of m |
|--|--------------|
| Very smooth cement surfaces or planed boards.. | 0.06 |
| Concrete, well-laid brick, unplanned boards..... | 0.16 |
| Ashlar, good rubble masonry, poor brickwork... | 0.46 |
| Earth beds in perfect condition..... | 0.85 |
| Earth beds in ordinary condition..... | 1.30 |
| Earth beds in bad condition covered with débris | 1.75 |

Kutter's Formula

The velocity of flow in a channel in feet per second equals

$$v = \frac{41.65 + \frac{0.00281}{s} + \frac{1.811}{n}}{1 + \left(41.65 + \frac{0.00281}{s}\right) \frac{n}{\sqrt{r}}} \sqrt{rs}$$

where r and s are as in Bazin's formula.

Values for n , the coefficient of roughness, are as follows:

| Character of channel | Value of n |
|---|--------------|
| Planned timber, glazed or enameled surfaces.... | 0.009 |
| Smooth clean cement..... | 0.010 |
| Unplanned timber, new well-laid brickwork.... | 0.012 |
| Smooth stonework, ordinary brickwork, iron... | 0.013 |
| Rough ashlar and good rubble masonry..... | 0.017 |
| Firm gravel..... | 0.020 |
| Earth in ordinary condition..... | 0.025 |
| Earth with stones, weeds, etc..... | 0.030 |
| Earth or gravel in bad condition..... | 0.035 |

FLOW OVER WEIRS

Contraction is **complete** when no edge of the weir is flush with the sides or bottom of the channel.

Contraction is **incomplete** when one or more sides of the weir have an interior border flush with the sides or bottom of the channel.

Francis' Formula

The flow over a weir in cubic feet per second is

$$Q = \frac{2}{3} [0.622 h (b - \frac{1}{10} nh) \sqrt{2gh}]$$

in which

h = head in feet of water on weir,

b = width of weir in feet,

$n = 2$ for complete contraction,

$n = 1$ for one end of weir flush with side of channel,

$n = 0$ for both ends of weir flush with sides of channel.

Bazin's Formula for Weirs

For overfall-weirs with end contractions suppressed, the flow in cubic feet per second is

$$Q = \frac{2}{3} n \left[1 + 0.55 \left(\frac{h}{p+h} \right)^2 \right] bh \sqrt{2gh}$$

in which the coefficient n has the value

$$n = 0.6075 + \frac{0.0148}{h}$$

h = depth in feet of water on weir,

b = width of weir in feet,

p = height in feet of the sill of the weir above the bottom of the channel of approach.

STRESSES IN PIPES AND CYLINDERS

Pressure in Pipes

The tensile stress in pounds per square inch in a pipe due to internal fluid pressure is:

For **thin pipes**, $p' = \frac{rp}{t}$

For **thick pipes or cylinders**,

$$p' = \frac{p(r+t)}{t}$$

r = inside radius of pipe in inches,

t = thickness of pipe in inches,

p = excess of internal over external pressure in pounds per square inch.

If S is the required factor of safety, then:

For **thin pipes**, $t = S \frac{rp}{P}$

For **thick pipes or cylinders**,

$$t = S \frac{rp}{P - pS}$$

in which r and p are as above, and

P = ultimate tensile strength of material of pipe (see Table of Strength of Materials).

Collapsing of Tubes

The collapsing pressure for Bessemer steel lap-welded tubes, for lengths greater than six diameters, is

$$\left. \begin{aligned}
 p &= 1000 \left(1 - \sqrt{1 - 1600 \frac{t^2}{d^2}} \right) \text{ when } \frac{t}{d} < 0.023 \\
 \text{or} \\
 p &= 86670 \frac{t}{d} - 1386 \quad \text{when } \frac{t}{d} > 0.023
 \end{aligned} \right\} \text{(Stewart's equations)}$$

in which

p = excess of external over internal pressure in pounds per square inch,

d = outside diameter of tube in inches,

t = thickness of tube wall in inches.

FLOW OF FLUIDS

Flow of Air Through Apertures

The weight of air in pounds discharged per second from a reservoir into the atmosphere is

$$\left. \begin{aligned}
 M &= 0.53 F \frac{p_1}{\sqrt{T_1}} \quad \text{when } p_1 > 2 p_a \\
 \text{or} \\
 M &= 1.06 F \sqrt{\frac{p_a (p_1 - p_a)}{T_1}} \quad \text{when } p_1 < 2 p_a
 \end{aligned} \right\} \text{Fliegner's equations}$$

p_1 = reservoir pressure in pounds per square inch absolute,

p_a = atmospheric pressure (14.7 pounds per square inch),

F = cross-section of aperture in square inches,

T_1 = absolute temperature of reservoir (degrees Fahr. + 459.6).

Flow of Steam Through Apertures

$$M = 0.0165 F p_1^{0.97} \quad (\text{Grashof's formula})$$

$$\left. \begin{aligned} M &= \frac{F p_1}{70} \quad \text{when } p_1 > \frac{5}{3} p_2 \\ M &= \frac{F p_2}{42} \sqrt{\frac{3(p_1 - p_2)}{2 p_2}} \quad \text{when } p_1 < \frac{5}{3} p_2 \end{aligned} \right\} \text{Napier's equations}$$

Grashof's formula applies when the final pressure is less than 58 per cent of the reservoir pressure.

- M = pounds of steam discharged per second,
 p_1 = reservoir pressure in pounds per square inch,
 p_2 = final pressure in pounds per square inch,
 F = cross-section of aperture in square inches.

Flow of Gas in Pipes

$$Q = 1000 \sqrt{\frac{d^5 h}{sl}} \quad (\text{Molesworth})$$

- Q = quantity of gas in cubic feet per hour,
 d = diameter of pipe in inches,
 l = length of pipe in yards,
 h = pressure in inches of water,
 s = specific gravity of gas relative to air.

Flow of Air in Pipes

$$v = 114.5 \sqrt{\frac{hd}{L}} \quad (\text{Hawksley})$$

- v = velocity in feet per second,
 h = head in inches of water,
 d = diameter of pipe in inches,

L = length of pipe in feet,

$$Q = \frac{\pi}{4} \frac{d^2}{144} v$$

Q = quantity in cubic feet per second.

Flow of Compressed Air in Pipes

$$Q = 217.5 \sqrt{\frac{pd^5}{rL}}$$

$$d = 0.1161 \sqrt[5]{\frac{LQ^2r}{p}} = 0.1161 \sqrt[5]{\frac{LQ_1^2}{pr}}$$

Q = volume in cubic feet per minute of compressed air, at 62° F.,

Q_1 = volume before compression, at 62° F.,

r = pressure in atmospheres,

p = difference in pressures in pounds per sq. inch, causing the flow,

d = diameter of pipe in inches,

L = length of pipe in feet.

Flow of Steam in Pipes

$$W = 87 \sqrt{\frac{w(p_1 - p_2)d^5}{L\left(1 + \frac{3.6}{d}\right)}} \quad (\text{Babcock})$$

W = weight of steam flowing in pounds per minute,
 w = density in pounds per cubic foot of the steam at the entrance to the pipe,

p_1 = pressure in pounds per square inch at the entrance,

p_2 = pressure at exit,

d = diameter in inches,

L = length of pipe in feet.

ELECTRICITY

OHMIC RESISTANCE

The resistance of a uniform electric conductor at 0° Centigrade is given by the formula:

$$R \text{ (in ohms)} = \rho \frac{L}{A}$$

L = length of conductor in inches,

A = cross-section in square inches,

ρ = resistivity of conductor at 0° C., values of which are given in the following table.

TABLE OF RESISTIVITIES

(Resistivity is the resistance in ohms between any two opposite faces of a 1 inch cube of the material)*

| Metal | Resistivity at 0° C. |
|-------------------------|------------------------|
| Aluminium (annealed) .. | 1.14×10^{-6} |
| Aluminium (commercial) | 1.05×10^{-6} |
| Aluminium bronze | 4.96×10^{-6} |
| Bismuth (compressed)... | 51.2×10^{-6} |
| Brass | 2.82×10^{-6} |
| Copper (drawn)..... | 0.637×10^{-6} |
| Copper (annealed)..... | 0.625×10^{-6} |
| German silver | 8.23×10^{-6} |
| Gold (annealed)..... | 0.803×10^{-6} |
| Iron (wrought)..... | 3.82×10^{-6} |
| Lead (compressed)..... | 7.68×10^{-6} |
| Magnesium..... | 1.72×10^{-6} |
| Mercury..... | 37.1×10^{-6} |
| Nickel (annealed)..... | 4.89×10^{-6} |
| Platinum (annealed).... | 3.53×10^{-6} |
| Silver (annealed)..... | 0.575×10^{-6} |
| Tin..... | 5.16×10^{-6} |
| Tungsten..... | $2. \times 10^{-6}$ |
| Zinc (pressed)..... | 2.28×10^{-6} |

* This definition applies to English units and to the numerical values given in the table. In general, resistivity is the resistance of a unit cube.

The resistance of a conductor at any temperature is

$$R_2 = R_1 \frac{(1 + \alpha t_2)}{(1 + \alpha t_1)}$$

in which

R_1 = known resistance at a temperature t_1 degrees Centigrade,

R_2 = required resistance at a temperature t_2 degrees Centigrade,

α = temperature coefficient of electrical resistance, the value of which is given for different metals in the following table.

TEMPERATURE COEFFICIENTS OF ELECTRICAL RESISTANCE

| Metal | Temp. coefficient (approx.) for 1° C. |
|---------------------------|---------------------------------------|
| Aluminium (commercial) .. | 0.00435 |
| Copper (annealed) | 0.00388 |
| German silver | 0.00036 |
| Gold (annealed) | 0.00365 |
| Iron (wrought) | 0.00463 |
| Mercury | 0.00072 |
| Platinum | 0.00247 |
| Silver | 0.00377 |
| Tungsten | 0.00570 |
| Zinc (pressed) | 0.00365 |

Note. — The temperature coefficient of a material is its increase in resistance for each degree Centigrade rise in temperature, and it is expressed as a decimal fraction of the resistance at 0° C.

DATA ON ANNEALED COPPER WIRE

| Gauge No. (B. & S.) | Diameter in mils | | | Cross-section of bare wire | | Resistance in ohms per 1000 feet | | Pounds per 1000 feet |
|------------------------|------------------|-----------------------|-----------------------|----------------------------|------------|----------------------------------|------------------------|----------------------|
| | Bare | Double cotton covered | | Circular mils | Sq. inches | Cold (25° C. = 77° F.) | Hot (65° C. = 149° F.) | |
| | | Single cotton covered | Double cotton covered | | | | | |
| 0000 | 460 | | | 212,000 | 0.166 | 0.0500 | 0.0577 | 641 |
| 0000 | 410 | | | 168,000 | 0.132 | 0.0630 | 0.0727 | 508 |
| 00 | 365 | | | 133,000 | 0.105 | 0.0795 | 0.0917 | 403 |
| 0 | 325 | | | 106,000 | 0.0829 | 0.100 | 0.116 | 319 |
| 1 | 289 | | | 83,700 | 0.0657 | 0.126 | 0.146 | 253 |
| 2 | 258 | | | 66,400 | 0.0521 | 0.159 | 0.184 | 201 |
| 3 | 229 | | | 52,600 | 0.0413 | 0.201 | 0.232 | 159 |
| 4 | 204 | | | 41,700 | 0.0328 | 0.253 | 0.292 | 126 |
| 5 | 182 | | | 33,100 | 0.0260 | 0.319 | 0.369 | 100 |
| 6 | 162 | 174 | | 26,300 | 0.0206 | 0.403 | 0.465 | 79.5 |
| 7 | 144 | 156 | | 20,800 | 0.0164 | 0.508 | 0.586 | 63.0 |
| 8 | 128 | 136 | | 16,500 | 0.0130 | 0.641 | 0.739 | 50.0 |
| 9 | 114 | 121 | 126 | 13,100 | 0.0103 | 0.808 | 0.932 | 39.6 |
| 10 | 102 | 108 | 112 | 10,400 | 0.00815 | 1.02 | 1.18 | 31.4 |
| 11 | 91 | 97 | 101 | 8,230 | 0.00647 | 1.28 | 1.48 | 24.9 |
| 12 | 81 | 87 | 91 | 6,530 | 0.00513 | 1.62 | 1.87 | 19.8 |
| 13 | 72 | 78 | 82 | 5,180 | 0.00407 | 2.04 | 2.36 | 15.7 |
| 14 | 64 | 70 | 74 | 4,110 | 0.00323 | 2.58 | 2.97 | 12.4 |
| 15 | 57 | 63 | 67 | 3,260 | 0.00256 | 3.25 | 3.75 | 9.86 |
| 16 | 51 | 56 | 59 | 2,580 | 0.00203 | 4.09 | 4.73 | 7.82 |
| 17 | 45 | 50 | 53 | 2,050 | 0.00161 | 5.16 | 5.96 | 6.20 |

DATA ON ANNEALED COPPER WIRE (Continued)

| Gauge No. (B. & S.) | Diameter in mils | | | Double cot- ton covered | Cross-section of bare wire | | Resistance in ohms per 1000 ft. | | Pounds per 1000 feet |
|------------------------|------------------|--------------------------|-------|----------------------------|----------------------------|------------|---------------------------------|--------------------------|-------------------------|
| | Bare | Single cotton covered | | | Circular mils | Sq. inches | Cold (25° C. =77° F.) | Hot (65° C. =149° F.) | |
| | | | | | | | | | |
| 18 | 40 | 45 | 48 | 1620 | 0.00128 | 6.51 | 7.51 | 4.92 | |
| 19 | 36 | 39 | 43 | 1290 | 0.00101 | 8.21 | 9.48 | 3.90 | |
| 20 | 32 | 36 | 40 | 1020 | 0.000802 | 10.4 | 11.9 | 3.09 | |
| 21 | 28.5 | 32.5 | 36.5 | 810 | 0.000636 | 13.1 | 15.1 | 2.45 | |
| 22 | 25.3 | 29.0 | 33.0 | 642 | 0.000505 | 16.5 | 19.0 | 1.94 | |
| 23 | 22.6 | 26.6 | 30.6 | 509 | 0.000400 | 20.8 | 24.0 | 1.54 | |
| 24 | 20.1 | 24.1 | 28.1 | 404 | 0.000317 | 26.2 | 30.2 | 1.22 | |
| 25 | 17.9 | 21.9 | 25.9 | 320 | 0.000252 | 33.0 | 38.1 | 0.970 | |
| 26 | 15.9 | 19.9 | 23.9 | 254 | 0.000200 | 41.6 | 48.0 | 0.769 | |
| 27 | 14.2 | 18.2 | 22.2 | 202 | 0.000158 | 52.5 | 60.6 | 0.610 | |
| 28 | 12.6 | 16.6 | 20.6 | 160 | 0.000126 | 66.2 | 76.4 | 0.484 | |
| 29 | 11.3 | 15.3 | 19.3 | 127 | 0.0000995 | 83.4 | 96.3 | 0.384 | |
| 30 | 10.0 | 14.0 | 18.0 | 101 | 0.0000789 | 105 | 121 | 0.304 | |
| 31 | 8.9 | 12.9 | 16.9 | 79.7 | 0.0000626 | 133 | 153 | 0.241 | |
| 32 | 8.0 | 11.9 | 15.9 | 63.2 | 0.0000496 | 167 | 193 | 0.191 | |
| 33 | 7.1 | 11.1 | 15.1 | 50.1 | 0.0000394 | 211 | 243 | 0.152 | |
| 34 | 6.3 | 10.3 | 14.3 | 39.8 | 0.0000312 | 266 | 307 | 0.120 | |
| 35 | 5.6 | 9.6 | 13.6 | 31.5 | 0.0000248 | 335 | 387 | 0.0954 | |
| 36 | 5.0 | 8.5 | 12.0 | 25.0 | 0.0000196 | 423 | 488 | 0.0757 | |
| 37 | 4.5 | | | 19.8 | 0.0000156 | 533 | 616 | 0.0600 | |
| 38 | 4.0 | | | 15.7 | 0.0000123 | 673 | 776 | 0.0476 | |
| 39 | 3.5 | | | 12.5 | 0.0000098 | 848 | 979 | 0.0377 | |
| 40 | 3.1 | | | 9.9 | 0.0000078 | 1070 | 1230 | 0.0299 | |

DATA ON ANNEALED COPPER WIRE

| Gauge No. (B. & S.) | Diameter in mils | | | Cross-section of bare wire | | Resistance in ohms per 1000 feet | | Pounds per 1000 feet |
|------------------------|------------------|-----------------------|-----------------------|----------------------------|------------|----------------------------------|------------------------|----------------------|
| | Bare | Double cotton covered | | Circular mils | Sq. inches | Cold (25° C. = 77° F.) | Hot (65° C. = 149° F.) | |
| | | Single cotton covered | Double cotton covered | | | | | |
| 0000 | 460 | | | 212,000 | 0.166 | 0.0500 | 0.0577 | 641 |
| 0000 | 410 | | | 168,000 | 0.132 | 0.0630 | 0.0727 | 508 |
| 00 | 365 | | | 133,000 | 0.105 | 0.0795 | 0.0917 | 403 |
| 0 | 325 | | | 106,000 | 0.0829 | 0.100 | 0.116 | 319 |
| 1 | 289 | | | 83,700 | 0.0657 | 0.126 | 0.146 | 253 |
| 2 | 258 | | | 66,400 | 0.0521 | 0.159 | 0.184 | 201 |
| 3 | 229 | | | 52,600 | 0.0413 | 0.201 | 0.232 | 159 |
| 4 | 204 | | | 41,700 | 0.0328 | 0.253 | 0.292 | 126 |
| 5 | 182 | | | 33,100 | 0.0260 | 0.319 | 0.369 | 100 |
| 6 | 162 | | | 26,300 | 0.0206 | 0.403 | 0.465 | 79.5 |
| 7 | 144 | | 174 | 20,800 | 0.0164 | 0.508 | 0.586 | 63.0 |
| 8 | 128 | | 156 | 16,500 | 0.0130 | 0.641 | 0.739 | 50.0 |
| 9 | 114 | | 141 | 13,100 | 0.0103 | 0.806 | 0.932 | 39.6 |
| 10 | 102 | | 126 | 10,400 | 0.00815 | 1.02 | 1.18 | 31.4 |
| 11 | 91 | | 112 | 8,230 | 0.00647 | 1.28 | 1.48 | 24.9 |
| 12 | 81 | | 101 | 6,530 | 0.00513 | 1.62 | 1.87 | 19.8 |
| 13 | 72 | | 91 | 5,180 | 0.00407 | 2.04 | 2.36 | 15.7 |
| 14 | 64 | | 82 | 4,110 | 0.00323 | 2.58 | 2.97 | 12.4 |
| 15 | 57 | | 74 | 3,260 | 0.00256 | 3.25 | 3.75 | 9.86 |
| 16 | 51 | | 67 | 2,580 | 0.00203 | 4.09 | 4.73 | 7.82 |
| 17 | 45 | | 59 | 2,050 | 0.00161 | 5.16 | 5.96 | 6.20 |
| | | | 53 | | | | | |

DATA ON ANNEALED COPPER WIRE (Continued)

| Gauge No. (B. & S.) | Diameter in mils | | | Cross-section of bare wire | | Resistance in ohms per 1000 ft. | | | Pounds per 1000 feet |
|------------------------|------------------|--------------------------|----------------------------|----------------------------|------------|---------------------------------|---------------------------|--------|-------------------------|
| | Bare | Single cotton covered | Double cot- ton covered | Circular mils | Sq. inches | Cold (25° C. = 77° F.) | Hot (65° C. = 149° F.) | | |
| | | | | | | | Hot (65° C. = 149° F.) | | |
| 18 | 40 | 45 | 48 | 1620 | 0.00128 | 6.51 | 7.51 | 4.92 | |
| 19 | 36 | 39 | 43 | 1290 | 0.00101 | 8.21 | 9.48 | 3.90 | |
| 20 | 32 | 36 | 40 | 1020 | 0.000602 | 10.4 | 11.9 | 3.09 | |
| 21 | 28.5 | 32.5 | 36.5 | 810 | 0.000636 | 13.1 | 15.1 | 2.45 | |
| 22 | 25.3 | 29.0 | 33.0 | 642 | 0.000505 | 16.5 | 19.0 | 1.94 | |
| 23 | 22.6 | 26.6 | 30.6 | 509 | 0.000400 | 20.8 | 24.0 | 1.54 | |
| 24 | 20.1 | 24.1 | 28.1 | 404 | 0.000317 | 26.2 | 30.2 | 1.22 | |
| 25 | 17.9 | 21.9 | 25.9 | 320 | 0.000252 | 33.0 | 38.1 | 0.970 | |
| 26 | 15.9 | 19.9 | 23.9 | 254 | 0.000200 | 41.6 | 48.0 | 0.769 | |
| 27 | 14.2 | 18.2 | 22.2 | 202 | 0.000158 | 52.5 | 60.6 | 0.610 | |
| 28 | 12.6 | 16.6 | 20.6 | 160 | 0.000126 | 66.2 | 76.4 | 0.484 | |
| 29 | 11.3 | 15.3 | 19.3 | 127 | 0.0000995 | 83.4 | 96.3 | 0.384 | |
| 30 | 10.0 | 14.0 | 18.0 | 101 | 0.0000789 | 105 | 121 | 0.304 | |
| 31 | 8.9 | 12.9 | 16.9 | 79.7 | 0.0000626 | 133 | 153 | 0.241 | |
| 32 | 8.0 | 11.9 | 15.9 | 63.2 | 0.0000496 | 167 | 193 | 0.191 | |
| 33 | 7.1 | 11.1 | 15.1 | 50.1 | 0.0000394 | 211 | 243 | 0.152 | |
| 34 | 6.3 | 10.3 | 14.3 | 39.8 | 0.0000312 | 266 | 307 | 0.120 | |
| 35 | 5.6 | 9.6 | 13.6 | 31.5 | 0.0000248 | 335 | 387 | 0.0954 | |
| 36 | 5.0 | 8.5 | 12.0 | 25.0 | 0.0000196 | 423 | 488 | 0.0757 | |
| 37 | 4.5 | | | 19.8 | 0.0000156 | 533 | 616 | 0.0600 | |
| 38 | 4.0 | | | 15.7 | 0.0000123 | 673 | 776 | 0.0476 | |
| 39 | 3.5 | | | 12.5 | 0.0000098 | 848 | 979 | 0.0377 | |
| 40 | 3.1 | | | 9.9 | 0.0000078 | 1070 | 1230 | 0.0299 | |

Ohm's Law

$$I = \frac{E}{R} \quad R = \frac{E}{I}$$

or $E = IR$

I = current in amperes,

E = electromotive force in volts,

R = resistance in ohms.

The proper size of wire in circular mils for any direct current circuit on a two-wire system consisting of copper conductors is given by the formula:

$$\text{c.m.} = \frac{10.8 \times 2 d \times I}{E}$$

or if the resistance is required,

$$r = \frac{E}{2 d \times I}$$

where

r = resistance per foot of wire in ohms,

E = volts drop in line,

I = total line current in amperes,

d = distance from source to load in feet,

c.m. = cross-section of conductor in circular mils.

Resistance of Circuits

The resultant of several resistances in **series** equals

$$R = r_1 + r_2 + r_3 + \dots$$

where r_1, r_2, r_3 , etc., are the separate resistances.

The resultant of several resistances in **parallel** or **multiple** is given by the relation:

$$\frac{1}{R} = \frac{1}{r_1} + \frac{1}{r_2} + \frac{1}{r_3} + \dots$$

R is the total or combined resistance; and r_1, r_2, r_3 , etc., are the separate resistances.

Power and Energy in Direct Current Circuits

The power in watts expended in a resistance is

$$P = EI = I^2R$$

E = electromotive force in volts,

I = current in amperes,

R = resistance in ohms.

The **energy** transformed into heat in a time t seconds is

$$\epsilon = EIt = I^2Rt$$

when the current, I , is constant; or, if the current is variable, energy equals

$$\epsilon = \int_{t_1}^{t_2} i^2R dt$$

where i is the instantaneous value of the current, expressed as a function of t .

The **power** in any two-wire direct current circuit is

$$P \text{ (in watts)} = EI$$

where E is the volts between the terminals of the circuit and I is the current in amperes.

MOTORS AND GENERATORS

The **frequency** in cycles per second is given by the relation:

$$f = \frac{\text{R.P.M.}}{60} \times \frac{P}{2}$$

R.P.M. = speed in revolutions per minute,

P = number of poles.

Equations of Direct Current Motor

The armature current of a motor, during starting, is

$$I_a = \frac{E - e}{R_a + R_x}$$

in which

E = impressed voltage,

e = counter-electromotive force,

R_a = armature resistance in ohms,

R_x = resistance of grid or rheostat in series with armature.

At full speed,

$$I_a = \frac{E - e}{R_a}$$

$$e = K\phi f$$

$$E = I_a R_a + e = I_a R_a + K\phi f$$

$$I_a = \frac{E - K\phi f}{R_a}$$

$$f = \frac{E - I_a R_a}{K\phi}$$

f = frequency in cycles per second,*

ϕ = total field flux in magnetic lines, cutting armature conductors,

K = constant for any given machine. Its value is

$\frac{4t}{10^8}$, where t is the number of armature turns in series.

* Frequency, in the case of a direct current machine, refers to the frequency of alternation in the armature windings, not, of course, in the external circuit.

Equations of Direct Current Generator

$$E = e - I_a R_a$$

e = generated voltage,

E = terminal voltage,

I_a = armature current in amperes,

R_a = armature resistance in ohms.

$$I_a = \frac{E}{R}$$

R = resistance of load in ohms.

$$E = RI_a$$

$$e = E + I_a R_a = I_a (R + R_a)$$

Torque

The torque of a dynamo in foot-pounds equals

$$T = KI\phi$$

where

ϕ = total field flux in magnetic lines, cutting armature conductors,

I = armature current in amperes,

K = constant term for any given dynamo. Its value

is $K = \frac{2.348}{10^9} tP$, t being the number of arma-

ture turns in series, and P the total number of poles.

The torque of a motor in terms of the horsepower is

$$T = \frac{33,000 \text{ H.P.}}{2\pi n}$$

or solving for horsepower,

$$\text{H.P.} = \frac{2\pi Tn}{33,000} = \frac{2\pi RFn}{33,000}$$

n = number of revolutions per minute,

T = torque in foot-pounds,

R = radius of pulley in feet,

F = turning force in pounds.

Induced Voltage

$$e = - \frac{N}{10^8} \frac{d\phi}{dt} \text{ volts}$$

N = number of turns.

If the turns cut across a uniform field, at right angles to the lines of force, then $\frac{d\phi}{dt}$ equals the number of lines cut per second. Otherwise, $\frac{d\phi}{dt}$ is the first derivative of ϕ in respect to t , ϕ being expressed as a function of t .

The **effective voltage** induced in the windings of a generator, motor, or transformer, etc., is given by the relation:

$$E = \frac{\sqrt{2} \pi f n \phi}{10^8} = \frac{4.44 f n \phi}{10^8} \text{ volts}$$

This formula is generally quite accurate, being derived on the assumption of uniform flux distribution.

f = frequency in cycles per second,

ϕ = total number of lines of magnetic force,

n = effective number of turns. If all the turns are grouped in one coil, then n equals the total number of turns. Otherwise, if the winding is distributed over k electrical degrees (as in the armature of a motor or generator), then

$$\text{the effective number of turns is } n = N \frac{\sin\left(\frac{k}{2}\right)}{\frac{k}{2}},$$

N being the total number of turns.

The **average induced voltage** of a dynamo is

$$E = \frac{4fn\phi}{10^8} \text{ volts}$$

where n is the number of armature turns in series.

Inductance

Inductance, L , is the number of interlinkages of flux with turns, per unit current,

$$L \text{ (henrys)} = \frac{N\phi}{10^8 I}$$

in which

N = number of turns,

I = current in amperes,

ϕ = number of lines of magnetic force interlinking with the turns.

The **theoretical unit** of inductance is the centimeter.

The **practical unit** of inductance is the henry, which equals 10^9 centimeters.

The **counter-electromotive force** in an inductive circuit is

$$e = -L \frac{di}{dt}$$

provided the inductance, L , is constant.

The **total voltage** consumed by an inductive circuit

$$E = ir + L \frac{di}{dt}$$

the inductance, L , being constant.

r is the resistance of the circuit in ohms, and $\frac{di}{dt}$ is the first derivative of i with respect to t , the current i being expressed as a function of t .

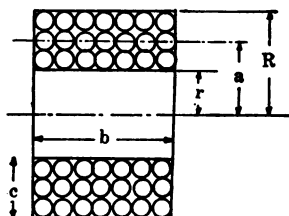
The inductance in henrys of an air-core circular coil is

$$L = \frac{0.366 \left(\frac{l}{1000} \right)^2}{b + c + R} \times F' F''$$

$$F' = \frac{10b + 12c + 2R}{10b + 10c + 1.4R}$$

$$F'' = 0.5 \log_{10} \left(100 + \frac{14R}{2b + 3c} \right)$$

l = length of conductor in feet.¹



All other dimensions are in inches and as indicated in the diagram.

The inductance, L , of a concentric cable in henrys per 1000 feet is

$$L = \frac{3.048}{10^5} \times$$

$$\left\{ \frac{1}{2} + 4.6 \log_{10} \frac{R}{r} + \frac{4.6 R_0^4}{(R_0^2 - R^2)^2} \log_{10} \frac{R_0}{R} - \frac{1}{2} \frac{3 R_0^2 - R^2}{(R_0^2 - R^2)} \right\}$$

where

r = radius of inner metallic conductor,

R = distance from center of cable to the inner surface of the outer metallic conductor,

R_0 = distance from center of cable to the outer surface of the outer metallic conductor.

The values of r , R , and R_0 must be expressed in the same units.

The **total inductance**, L , of a **two-wire transmission circuit** in henrys per 1000 feet is

$$L = \frac{3.048}{10^5} \left\{ 9.2 \mu \log_{10} \frac{D-r}{r} + \mu_1 \right\}$$

where

μ_1 = permeability of the metal conductor; for copper,

$$\mu_1 = 1,$$

μ = permeability of medium separating wires; for air, $\mu = 1$,

D = distance between the two lines, measured from center to center,

r = radius of conductor, in same unit as D .

Capacity

The **unit of capacity** is the **farad**. Since the farad is very large, the **microfarad**, which is one-millionth of a farad, is used as the practical unit. The **theoretical unit** of capacity is the centimeter, 9×10^{11} centimeters being equal to 1 farad.

The **charge** of a condenser, Q , is measured in ampere-seconds or coulombs, and may be calculated by the formula:

$$Q = CE$$

from which

$$C = \frac{Q}{E}$$

and

$$E = \frac{Q}{C}$$

where

C = capacity in farads,

E = potential across the terminals of the condenser in volts.

The capacity of a plate condenser is

$$C = \frac{2248 KA}{d \times 10^{10}} \text{ microfarads}$$

where

A = total area in square inches of **all** the dielectric sheets separating the condenser plates,

d = average thickness in inches of one sheet of the dielectric,

K = inductivity of the dielectric, average values of which are given in the following table for different materials.

| Materials | Inductivity K |
|-----------------------------|-----------------|
| Air (at standard pressure). | 1.00 |
| Manilla paper..... | 1.50 |
| Paraffin, solid..... | 2.00 |
| Ebonite..... | 2.50 |
| India rubber..... | 2.50 |
| Shellac..... | 3.00 |
| Oil..... | 3.00 |
| Glass..... | 3.10 |
| Mica..... | 6.00 |

Condensers in Parallel. When two or more condensers are connected in parallel, the resultant capacity, C , equals the sum of the separate capacities, thus

$$C = C_1 + C_2 + C_3 + \dots$$

Condensers in Series. When two or more condensers of capacities C_1, C_2, C_3 , etc., are connected in series, the resultant capacity is given by the formula:

$$C = \frac{1}{\frac{1}{C_1} + \frac{1}{C_2} + \frac{1}{C_3} + \dots}$$

The **capacity**, C , of a **concentric cable** per 1000 feet in microfarads is

$$C = \frac{7.37}{1000 \log_{10} \frac{\rho_0}{\rho}}$$

in which

ρ = radius of inner metallic conductor,

ρ_0 = distance from center of cable to the inner surface of the outer metallic conductor, in the same unit as ρ .

The **capacity**, C , of a **two-wire transmission line** per 1000 feet in microfarads is given approximately by the formula:

$$C = \frac{3.68}{1000 \log_{10} \frac{D-r}{r}}$$

if the lines are not close to the ground.

D = distance between the two wires of the transmission line, measured from center to center,

r = radius of conductor, in same unit as D .

The **differential equations** of a **condenser** are

$$dq = i dt$$

$$q = \text{charge} = \int i dt$$

$$dq = c de$$

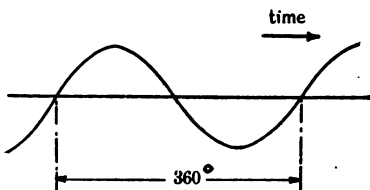
$$i = c \frac{de}{dt}$$

Alternating Current Circuits

The shape of the voltage or current wave produced by an alternator is, in general, nearly that of a **sine curve**. Alternating current calculations are, therefore, usually worked out on this assumption.

The number of cycles or complete waves per second is the **frequency** of the current, and the time required for the current to complete one cycle is a **period**.

The **average value** of the current or voltage is the average of all the



ordinates of the curve of one half-wave. The **effective value** of an alternating current or voltage is the square root of the sum of the squares of the instantaneous values of a half-wave.

If E is the maximum voltage of a half-cycle of a sine wave,

$$\text{average voltage} = \frac{2}{\pi} E = 0.636 E$$

$$\text{effective voltage} = \frac{1}{\sqrt{2}} E = 0.707 E$$

Similarly, if the maximum current is I ,

$$\text{average current} = \frac{2}{\pi} I = 0.636 I$$

$$\text{effective current} = \frac{1}{\sqrt{2}} I = 0.707 I$$

When the voltage reaches a definite value in the cycle sooner than the current reaches its corresponding value, the voltage and current are **out of phase** with each other; the voltage is said to be **leading**, and the current to be **lagging**. Phase difference is always expressed in degrees; a complete cycle equals 360 degrees.

Alternating Voltage and Current

$$I = \frac{E}{Z} \quad Z = \frac{E}{I}$$

or
$$E = IZ$$

I = current in amperes,

E = electromotive force in volts,

Z = impedance in ohms.

Impedance and Reactance

r = resistance in ohms

x = reactance in ohms

z = impedance in ohms

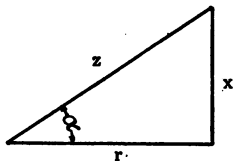
The relation between resistance, reactance, and impedance is the same as that between the three sides of a right triangle.

$$r = z \cos \alpha$$

$$x = z \sin \alpha$$

$$\alpha = \tan^{-1} \frac{x}{r}$$

$$z = \sqrt{r^2 + x^2}$$



Inductive Circuits

The inductive reactance in ohms is

$$x_L = 2\pi fL$$

where f = frequency in cycles per second,

L = inductance in henrys.

The impedance in ohms is

$$z = \sqrt{r^2 + x_L^2} = \sqrt{r^2 + 4\pi^2 f^2 L^2}$$

Circuits having Capacity

The capacity reactance in ohms is.

$$x_C = -\frac{1}{2\pi fC}$$

where f = frequency in cycles per second,
 C = capacity in farads.

The impedance in ohms is

$$z = \sqrt{r^2 + x_C^2} = \sqrt{r^2 + \frac{1}{4\pi^2 f^2 C^2}}$$

Circuits having Inductance and Capacity

The reactance in ohms is

$$x = x_L + x_C = 2\pi fL - \frac{1}{2\pi fC}$$

The impedance in ohms equals

$$z = \sqrt{r^2 + (x_L + x_C)^2}$$

Vector Representation of Sine Waves

A sine wave of voltage or current may be represented by a vector, the magnitude or length of which is equal to the effective value of the sine wave. It is sometimes more convenient to let the length of the vector equal the maximum value of the sine wave. The vector is generally denoted by a capital letter, with a dot directly beneath it; it is expressed in terms of its rectangular components, which determine the magnitude of the vector and its direction relative to the coordinate axes. Thus, the vector \dot{E} is written

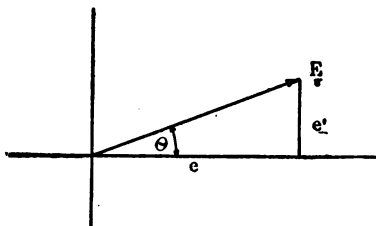
$$\dot{E} = e + j\dot{e}'$$

in which

$$j = \sqrt{-1}$$

where e denotes the horizontal or real component of the

vector, and e' the vertical or imaginary component. The imaginary unit, j , in the above equation, merely denotes the direction of measurement of e' .



The magnitude of E is

$$E = \sqrt{e^2 + e'^2}$$

and the angle θ which the vector E makes with the horizontal axis is

$$\theta = \tan^{-1} \frac{e'}{e}$$

The angle in degrees between two vectors is the **phase difference** between the two sine waves which the vectors represent.

In vector notation, the **impedance** is

$$Z = r + jx$$

and its magnitude is

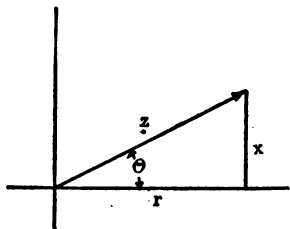
$$Z = \sqrt{r^2 + x^2}$$

The **admittance** is

$$Y = \frac{1}{Z} = \frac{1}{r + jx} = \frac{r}{Z^2} - j \frac{x}{Z^2} = g + jb$$

where $g = \frac{r}{Z^2} = \text{conductance,}$

$$b = -\frac{x}{Z^2} = \text{susceptance.}$$



The current equals

$$\underline{I} = \frac{\underline{E}}{\underline{Z}} = \underline{E} \underline{Y} = (e + je') \left(\frac{r}{Z^2} - j \frac{x}{Z^2} \right) = i + ji'$$

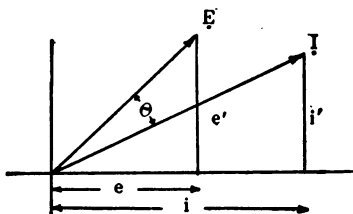
and the voltage is

$$\underline{E} = \underline{I} \underline{Z} = (i + ji') (r + jx) = e + je'$$

Power in Alternating Current Circuits

If the effective voltage and current are represented by the vectors

$$\begin{aligned} \underline{E} &= e + je' \\ \underline{I} &= i + ji' \end{aligned}$$



the real power is

$$W = ei + e'i' = EI \cos \theta$$

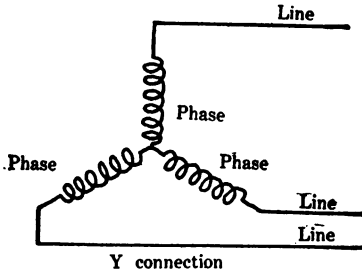
the wattless power is

$$W_s = e'i - ei' = EI \sin \theta$$

the volt-amperes equals EI .

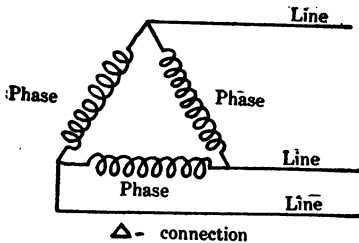
The power-factor is the cosine of the angle between the voltage and current vectors,

$$\text{power-factor} = \cos \theta = \frac{ei + e'i'}{EI}$$

Balanced Three-phase Circuits E = volts between lines e = volts per phase I = current in each line i = current in each phase

For Y-connections,

$$E = e\sqrt{3}; \quad e = \frac{E}{\sqrt{3}}; \quad \text{and} \quad I = i$$



For Δ-connections,

$$E = e; \quad I = i\sqrt{3}; \quad \text{and} \quad i = \frac{I}{\sqrt{3}}$$

In either case, for non-inductive load, the power in watts is

$$W = \sqrt{3} EI$$

If the load is inductive, then the power is

$$W = \sqrt{3} EI \cos \theta$$

where $\cos \theta$ is the power-factor of the phase.

MAGNETISM

Equations of Magnetic Circuits

F = attractive or repellent force in dynes,

mmf = magnetomotive force in ampere turns,

N = number of turns,

I = current in amperes,

β = density in magnetic lines per square centimeter,

ϕ = total number of lines of flux,

A = cross-section of magnetic path in square centimeters,

μ = permeability,

H = intensity of field,

l = length of magnetic circuit in centimeters,

ρ = reluctance,

m = pole strength,

r = distance between poles.

$$\phi = \frac{0.4 \pi NI}{\rho}$$

$$\rho = \frac{l}{\mu A}$$

$$\phi = \frac{0.4 \pi NI \mu A}{l}$$

$$\beta = \frac{\phi}{A}$$

$$\beta = \frac{0.4 \pi NI \mu}{l}$$

$$mmf = 0.4 \pi NI$$

$$\mu = \frac{\beta}{H}$$

Magnets and Magnetic Fields

$$F = mH$$

$$F = \frac{mm'}{\mu r^2}$$

$$\phi = 4\pi m$$

The attractive force in pounds exerted by a two pole magnet is $P = \frac{SB^2}{72,134,000}$, where S is the total area of both pole faces in square inches, and B is the density in magnetic lines per square inch.

The ampere-turns required to maintain a flux density of B lines per square inch in an air gap is $IN = 0.313 Bl$, in which l is the length of the gap in inches.

Hysteresis Loss

The power in watts lost in hysteresis is

$$W = k \frac{fVB^{1.6}}{10^7}$$

f = frequency in cycles per second,

V = volume of iron in cubic inches,

B = magnetic density in lines per square inch,

k = empirical constant, values of which are given in the following table.

| Character of iron | Value of k |
|-------------------------|-------------------|
| Silicon steel..... | 0.0006 to 0.00075 |
| Annealed sheet iron.... | 0.0008 to 0.0011 |
| Cast steel..... | 0.010 to 0.012 |
| Cast iron..... | 0.013 to 0.017 |

Eddy Current Loss

The power in watts lost due to eddy currents in iron or steel laminations is approximately

$$W = \frac{0.00135}{10^7} f^2 l^2 B^2 V$$

f = frequency in cycles per second,

l = average thickness of lamination in inches,

B = magnetic density in lines per square inch,

V = volume of iron in cubic inches.

This formula holds for ordinary temperatures, and if the thickness of the lamination is not greater than 0.025 inch. In silicon steel, the eddy current loss is approximately $\frac{1}{3}$ of that given above.

STANDARD SATURATION CURVES

B = density in lines per square inch

$AT/in.$ = ampere-turns per inch

Values of ampere-turns per inch for densities not included in the following tables may be determined approximately by interpolation. Thus, the $AT/in.$ for silicon steel for $B/sq.in. = 65,500$ is

$$AT/in. = 4.5 + \frac{5500}{10,000} (6.4 - 4.5) = 5.5 \text{ (approx.)}$$

| SILICON STEEL | | ANNEALED SHEET IRON | |
|------------------|---------------|---------------------|---------------|
| Saturation curve | | Saturation curve | |
| <i>B</i> | <i>AT/in.</i> | <i>B</i> | <i>AT/in.</i> |
| 30,000 | 2.1 | 30,000 | 4 |
| 40,000 | 2.7 | 40,000 | 4.4 |
| 50,000 | 3.4 | 50,000 | 5 |
| 60,000 | 4.5 | 60,000 | 9 |
| 70,000 | 6.4 | 70,000 | 12 |
| 80,000 | 10 | 80,000 | 20 |
| 90,000 | 23 | 90,000 | 33 |
| 100,000 | 35 | 100,000 | 60 |
| 110,000 | 100 | | |
| 120,000 | 225 | | |
| 130,000 | 520 | | |
| 135,000 | 1000 | | |
| 140,000 | 2200 | | |
| 145,000 | 3770 | | |
| 150,000 | 5330 | | |
| 155,000 | 6900 | | |

| CAST STEEL | | CAST IRON | |
|------------------|---------------|------------------|---------------|
| Saturation curve | | Saturation curve | |
| <i>B</i> | <i>AT/in.</i> | <i>B</i> | <i>AT/in.</i> |
| 50,000 | 11 | 5,000 | 8 |
| 60,000 | 15 | 10,000 | 12 |
| 70,000 | 20 | 15,000 | 17 |
| 80,000 | 29.5 | 20,000 | 23 |
| 90,000 | 50 | 25,000 | 30 |
| 100,000 | 105 | 30,000 | 43 |
| 105,000 | 165 | 35,000 | 60 |
| | | 40,000 | 85 |
| | | 45,000 | 110 |
| | | 50,000 | 145 |
| | | 55,000 | 190 |

AERONAUTICS

Balloons

For either rigid or non-rigid airships, in vertical equilibrium,

$$W + Vd' = Vd, \quad W = V(d - d')$$

where W = gross weight in lb., exclusive of gas,

V = volume of gas bag, cu. ft.,

d, d' = densities of external air and internal gas, lbs. per cu. ft.

If P = absolute pressure, lb. per sq. ft.: T = absolute temperature (Fahr. temp. + 460), m = molecular weight of gas, then,

$$d' = P'm' \div 1544 T', \quad d = P \div 53.36 T,$$

$$W = V \left(\frac{P}{53.36 T} - \frac{P'm'}{1544 T'} \right)$$

The lift (W) per cu. ft. of gas is $\frac{W}{V} = \frac{P}{53.36 T} - \frac{P'm'}{1544 T'}$, say $A - Bm'$, where P, T, P' and T' are standardized. The ratio of lifts per cu. ft. of two gases is then $r = (A - Bm_2') \div (A - Bm_1')$, under like conditions. The ratio of lifts per lb. of two gases under like conditions is $r \frac{m_1'}{m_2'}$.

Values of P and T are determined chiefly by the altitude (see page 202). Up to 10,000 ft., P decreases about 70 lb. from its sea-level value per 1000 ft. of

ascent. The value of T also decreases, and the seasonal variations in T decrease as altitude increases. The excess of P' over P is usually equivalent only to a few ounces per sq. in. This excess determines the tearing stress in the fabric (see page 153) which may be of the order of 100 lb. per lineal inch. Values of T' and T will differ somewhat, although the gas bag is painted with a non-absorbent coating.

Altitude control is most simply accomplished by dropping ballast (decreasing W) or by venting gas. The range of control is then greatest at the start and dampens down gradually by leakage or by the use of control. *Ballonets* (air bags), pumped full when it is desired to descend, prolong the control. There is a maximum allowable altitude (corresponding with a definite value of P) for every assigned set of conditions.

Resistance to flight in dirigible balloons at sea-level is given by $R = K V_0^n \sqrt{A^n}$, where R is in lb., A = total surface area of gas bag, sq. ft., V_0 = speed, ft. per sec., n is around 1.9 and K around 0.000015 for usual shapes from 4 to 8 diameters long. This must be somewhat increased to cover resistance to forward motion of car, structure, etc., The power required (thrust h.p. at the propeller) is $RV_0 \div 550$.

For minor items of resistance, if S = projected area on a transverse plane, sq. ft., and V is in miles per hr., $R' = K' S V^2$, where K' has the following values at sea-level:

| | |
|-------------------------------|--------------|
| Smooth wires normal to air, | $K' = .0026$ |
| Cables normal to air, | $K' = .003$ |
| Average for wheels uncovered, | $K' = .002$ |
| covered, | $K' = .001$ |

Values of K , K' , R and R' are directly proportional to the atmospheric density.

Airplanes

Wing Characteristics:

$$L = d k_L AV^2, \quad D = d k_D AV^2,$$

where d = atmospheric density,

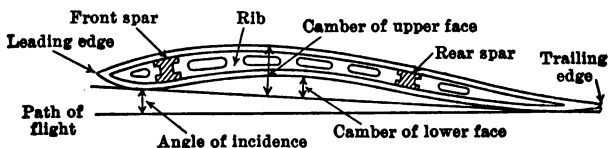
L = lift, in lb., at right angles with the flight path,

D = drift or wing resistance, in lb., parallel with the flight path,

A = area of wing, sq. ft.,

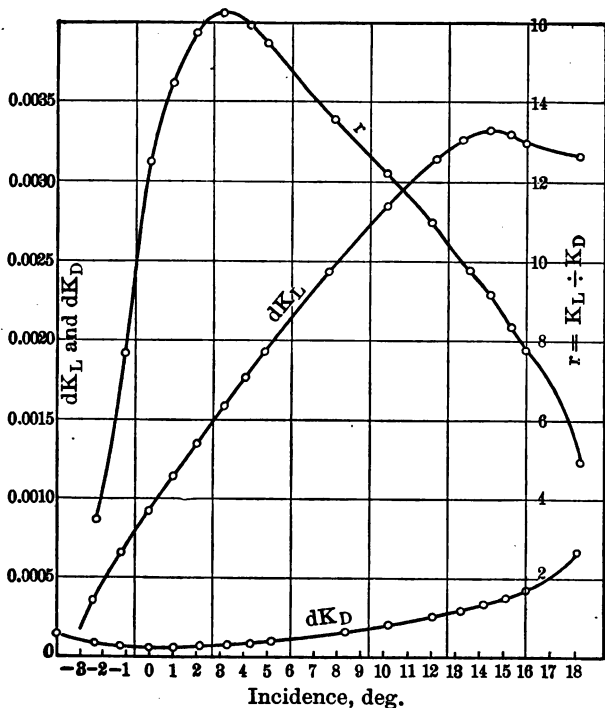
k_L and k_D = lift and drift coefficients.

Values of k_L and k_D depend on size, shape, camber, etc. They are determined by wind tunnel experiments. The chief factor in determining the values of the coefficients is the **incidence** of the bottom chord of the wing against the flight path. Lift and drift are com-



ponents of an approximately normal force acting at the **center of pressure**. This force is mainly a suction on the upper face. The position of the center of pressure varies with the incidence.

As indicated in the diagram, maximum values of k_L are around 0.043 and maximum values of $r = k_L/k_D$



around 16. There is positive lift at slight negative incidences. The incidence of maximum lift (usually around 14°) is called the **critical incidence** or **burble point**. It should not be approached too closely. In horizontal flight, $L = W$, the weight of the plane, and speeds increase as incidences decrease. **Minimum** speed is determined by maximum k_L and should be low for safe landing. Least resistance (wing resistance alone) is realized at incidences around 0°, but here the lift is low. The best ratio of lift to drift occurs

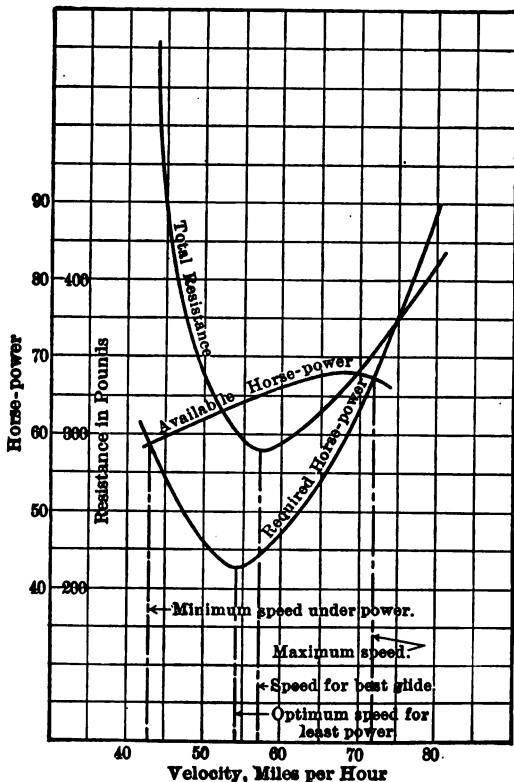
around 3° incidence and this determines the economical cruising speed, though lower incidences are generally used at maximum speeds.

If $L/A = W/A = l$ (the "wing loading"), $dV^2 = \frac{l}{k_L}$ and $\frac{D}{A} = \frac{l}{r}$. A given wing at a given incidence is subject to drift which is directly proportional to its loading. Also, since $V^2 = \frac{l}{d k_L}$, the landing speed is low (safe) for a given critical incidence when the wing loading is low.

Parasitic resistances (struts, bars, etc.) are determined as for balloons. Very roughly, $P = WV^2 \div 50,000$ lb. at sea-level, where P = total parasitic resistance, lb. Parasitic resistance varies directly as the density, but is practically independent of the incidence or wing area. Writing $P = d k_P V^2$, the total resistance is $R = D + P = d V^2 (A k_D + k_P)$. Parasitic resistance becomes very important at high speeds and low incidences. The ratio $\frac{L}{R}$ is commonly around 5 or 6 at usual incidences, where $\frac{L}{D}$ may be as high as 12 to 16. For a given plane, greatest distance of flight is realizable when R is a minimum. Since $\text{h.p.} = H = R V \div 375$, efficiency, which is proportional to $\frac{L}{R}$, is also proportional to $\frac{W V}{H}$, which fraction really expresses $\frac{\text{effect}}{\text{cause}}$.

The **h.p. required** has a definite value, for a given plane, corresponding with each speed or incidence. At a given incidence, the power to propel any par-

ticular plane varies directly as the atmospheric density: therefore inversely with the altitude. The diagram shows that minimum resistance occurs at a higher



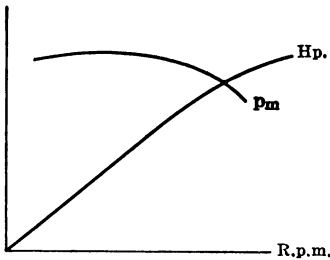
speed (lower incidence) than does minimum power required. The latter condition is the condition of maximum endurance (hours in the air). Since $W (= L)$ decreases during flight, the speeds of least power and

least resistance will vary: hence (whether the aim is greatest radius or greatest endurance) speed and incidence should vary during long horizontal flights.

During a **glide** at an angle θ with the horizontal, the plane is subject to forces R along the flight path and $L = R \tan \theta$, perpendicular to R , both due to its speed. A minimum gliding angle is reached, for a given plane, when R is a minimum. The incidence and speed for best glide are thus determined.

As to curve of **available horse-power** see page 188.

Engines. These are vertical, Vee, radial: stationary or rotary. Most are water-cooled. Radial stationary engines are



sometimes, and rotary engines always, air-cooled. Nearly all engines are four-cycle. Weights of present standard engines range somewhat beyond the limits 2 to $4\frac{1}{2}$ lb. per h.p.

Up to 16 cylinders or more are used. The 6-cylinder vertical, 8-cylinder 90° Vee and 12-cylinder 60° Vee, with 9-cylinder rotaries, are favorite assemblies. About 30 h.p. is the maximum size of cylinder: and this is reached only with water-cooling. Mean effective pressures, p_m , referred to the brake, reach 110 lb. in water-cooled and 80 lb. in air-cooled types. Speeds may be as high as 2000 R. P. M., but a reduction gear between engine and propeller is commonly employed for speeds exceeding 1600 R. P. M. Mean effective pressures (at sea-level) are constant over a considerable

speed range. Horse-power varies directly with the speed, up to a rather high limit. For one single-acting four-cycle cylinder of d in. diameter, and s in. stroke, at n R. P. M., brake h.p. = $0.7854 \, s n d^2 \dot{p}_m \div (24 \times 33,000)$.

Propellers are usually of wood, two-bladed. For similar propellers,

$$T = a n^2 D^4 d, \quad Q = b N^2 D^5 d = \frac{E d^2 s \dot{p}_m}{96}, \quad e = c,$$

where T = thrust, lb.,

Q = torque, lb. at 1 ft. radius,

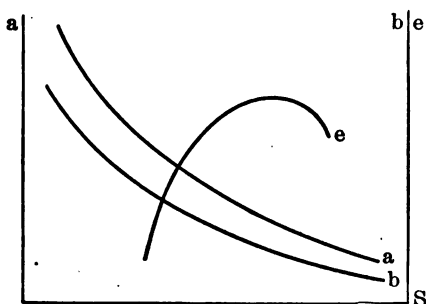
E = number of engine cylinders,

n = R. P. M.

D = diameter, ft.,

$$e = \text{propeller efficiency} = \frac{TV}{375} \div \frac{\pi Q n}{33,000}$$

a , b , and c = factors depending on the slip.



The values of a , b , and c are usually determined from wind tunnel experiments, and plotted on the base,

$S = \frac{V}{nD}$, which may be called the **effective pitch ratio**

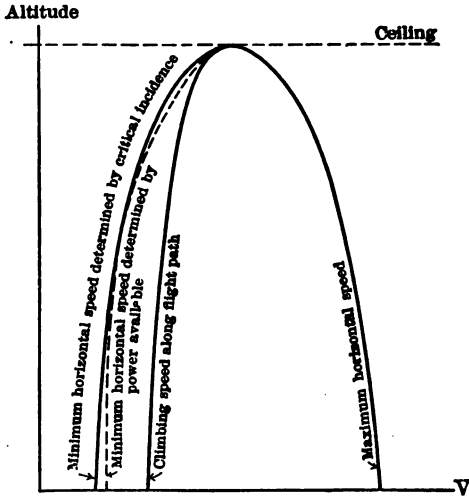
and is equal in homologous units to $p(1 - s) \div D$, where p = pitch, s = slip.

In a given wheel, D is constant. Hence for constant efficiency, $\frac{V}{n} = \text{constant}$. As will be shown, such constant ratio cannot be preserved: hence the propellor efficiency, which has a maximum value around 0.70, will seriously decrease at conditions other than those of the design.

Horizontal Power and Speed. Flying horizontally at sea-level, the engine and propellor torque remaining constant (because p_m is constant), $a_2 n_2^2 = a_1 n_1^2$ and n decreases as S decreases. Then $\frac{V_2}{V_1} = \frac{S_2 n_2}{S_1 n_1}$, or decreasing V leads to decreasing S and n , and (if the propellor was designed for a maximum value of e at the highest value of V) it also leads to decreasing e . Reduced plane speed decreases engine output because it decreases n . It may further decrease propellor output by decreasing e . These considerations explain the shape of the "available horse-power" curve of the diagram on page 185. The speed limits are determined by the intersections of this curve with that for "power required." For a given plane, a low loading lowers the latter curve and increases the speed range.

High Level Flight. Curves similar to that last mentioned may be plotted for various altitudes and corresponding densities. It will be found that the power required for horizontal flight at a high altitude is less than that at sea-level at a high velocity and greater at a low velocity. Minimum power required is greater and is realized at a greater velocity, as the altitude increases. The power available from the engine decreases. The indicated power varies in

almost direct proportion with the atmospheric density. The engine friction losses remain about constant. Hence the torque power falls off rather more rapidly than the density, as the altitude increases. This alone might be sufficient to decrease the speed range, with increasing altitude. As a matter of observed fact,



the maximum horizontal speed does decrease as the altitude increases. The value of n also decreases, though only slightly. Hence S decreases and in general e falls off, so that the "power available" curve is decidedly lowered at high altitudes. Eventually there is reached an altitude (**ceiling** or **absolute ceiling**) at which only one speed is possible and above which flight is impossible.

Climbing. The best condition for climbing is that at which there is the greatest surplus of power avail-

CHARACTERISTICS

| Service | Name | Form | Power | Nationality | Dimensions, ft. | | | Wing Area, sq. ft. |
|-------------|------------------|------|---------|-------------|-----------------|--------------|--------|--------------------|
| | | | | | Spread | Chord | Length | |
| Scout.... | Spad 13C1..... | B | TRA | F | 26.3 | .. | 20.4 | 215 |
| " | Martinsyde..... | B | ... | E | .. | .. | .. | 327 |
| " | Christmas Bullet | B | TRA | A | { 28.0 14.0 | { 5.0 2.5 | 21.0 | 170 |
| Combat... | De Haviland 9.. | B | TRA | E | 42.4 | 5.5 | 30.8 | 434 |
| Scout.... | Sopwith-Dolphin | .. | .. | E | .. | .. | .. | 263 |
| Combat... | Le Pere | B | TRA | A | 39.0 | 5.6 | 25.4 | 392 |
| " .. | Loening..... | M | TRA | A | 33.3 | 7.0 | .. | 239 |
| " .. | Curtiss 18-2.... | T | TRA | A | 31.9 | 3.5 | 23.3 | 309 |
| Observation | Albatross C3.... | B | TRA | G | { 38.8 36.7 | { 5.9 5.6 | 26.0 | 407 |
| Bomber... | Caproni..... | B | 2TRA,IP | I | 76.8 | 9.1 | .. | 1420 |
| " .. | Handley-Page... | B | TRA | A | 100.0 | 10.0 | .. | 1648 |
| " .. | Martin..... | B | TRA | A | 71.4 | 7.8 | .. | 1070 |
| Training... | Standard E1.... | B | TRA | A | 24.0 | 3.5 | .. | 153 |
| " .. | U. S. Std..... | B | TRA | A | 43.8 | 6.0 | .. | 455 |
| " .. | Curtiss R4 *.... | B | TRA | A | { 48.3 38.4 | { 6.3 | .. | 505 |
| Mail plane. | Standard..... | B | TRA | A | 31.4 | 6.0 | 26.6 | 337 |

B = biplane; TRA = tractor; F = French; A.I. = "Automotive Industries;" Age;" M = monoplane; Av. = "Aviation;" T = triplane; G = German; \mathcal{A} =

able over power required. The amounts of *power required* for horizontal flight at this condition do not vary much with the altitude, for a given plane. Hence, low weight per h.p. of engine capacity favors rapid climb. The excesses of power available will be found to decrease in a straight-line relation with the altitude. But $H' = Wc \div 33,000$, where H' = power available,

OF AIRPLANES

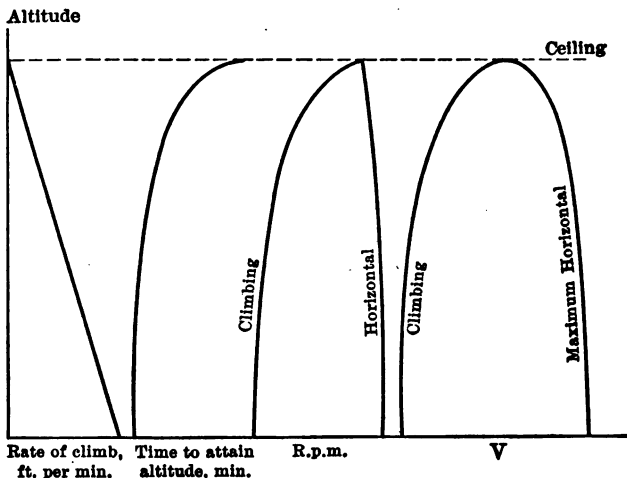
| Weights, lb. | | Speed | | Engine | | Loadings, lb. | | Ratio, Useful Weight ÷ Gross Weight | Reference |
|--------------|--------|---------------|---------------|----------------------|-------|---------------|-----------|-------------------------------------|---------------------|
| Gross | Useful | Miles per hr. | Altitude, ft. | Name | h. p. | Per sq. ft. | Per h. p. | | |
| 1815 | 556 | 130 | 6,500 | Hispano.... | 200 | 8.4 | 9.1 | 0.31 | A.I., Jan. 16, 1919 |
| 2289 | .. | 143 | 10,000 | Hispano.... | 300 | 7.0 | 7.5 | | |
| 2100 | 280 | 175 | S.L. | Hall-Scott, L6A..... | 200 | 12.3 | 10.5 | 0.13 | Fl., Feb. 13, 1919 |
| 3725 | .. | .. | .. | Lion..... | 420 | 8.5 | 8.8 | .. | Fl., Jan. 9, 1919 |
| 2358 | 792 | 140 | 10,000 | Hispano.... | 300 | 9.0 | 7.9 | 0.34 | Fl., Feb. 6, 1919 |
| 3655 | 1187 | 136 | S.L. | Liberty.... | 400 | 9.3 | 10.2 | 0.33 | A.A., Jan. 13, 1919 |
| 2368 | 1040 | 145 | S.L. | Hispano.... | 300 | 9.9 | 7.9 | 0.44 | Av., Jan. 15, 1919 |
| 2901 | 1076 | 151 | S.L. | Curtiss K12 | 400 | 9.4 | 7.3 | 0.37 | Av., Feb. 15, 1919 |
| 2790 | 960 | .. | .. | Mercedes... | 160 | 6.8 | 12.4 | 0.34 | AE., Mar. 6, 1918 |
| 9900 | 3300 | 91 | S.L. | Fiat..... | 750 | 7.0 | 13.2 | 0.33 | |
| 13700 | 5430 | 93 | S.L. | Liberty.... | 800 | 8.3 | 17.1 | 0.40 | |
| 9663 | 3801 | 119 | S.L. | Liberty.... | 800 | 9.0 | 12.0 | 0.39 | |
| 1144 | 316 | 100 | S.L. | Le Rhone... | 80 | 7.5 | 14.3 | 0.30 | |
| 1950 | 600 | 68 | S.L. | Curtiss..... | 90 | 4.3 | 21.7 | 0.31 | |
| 3242 | 1017 | 90 | S.L. | Curtiss..... | 200 | 6.4 | 15.9 | 0.31 | |
| 2400 | 834 | 100 | S.L. | Hispano I.. | 170 | 7.1 | 14.2 | 0.35 | Av., Jan. 1, 1919 |

E = English; A = American; S.L. = sea-level; Fl. = "Flight;" A.A. = "Aerial Aeronautics;" P = pusher; I = Italian.

* Also used as a 2-seater mailplane, when equipped with a 400 h.p. Liberty engine.

c = climb in ft. per min. Hence for a given plane, c bears a straight-line relation with the altitude. During the climb the plane speed (along its flight path) increases, the incidence being 6° to 8° at the start and usually 10° or 12° at the ceiling. The low speed at the start implies a low R. P. M. of the engine, which increases during the climb. The time-altitude

curve becomes horizontal at the ceiling. The **service ceiling** is that altitude at which the rate of climb becomes 100 ft. per min.



High ceiling is associated with rapid climbing. Both are favored by (a) low weight per h.p.; (b) low weight per sq. ft. of wing; (c) low parasitic resistance; (d) low engine friction.

Weights include structure, power plant, fuel and oil, crew and equipment. The two last mentioned rarely constitute over 35 per cent. of the total weight. The power plant weight includes radiator and water (for water-cooled engines), fuel and oil tanks, piping, and ignition apparatus. Water-cooled engines use about 0.55 lb. of fuel and 0.05 lb. of oil per h.p.-hr.: air-cooled, 50 per cent. more fuel and three times as much oil. On account of their lower weight per h.p., the latter engines have an advantage for short flights.

MEASUREMENT

English Weights and Measures

Length

| | | |
|--------------------------------|---|--------------|
| 1000 mils | = | 1 inch |
| 12 inches | = | 1 foot |
| 3 feet | = | 1 yard |
| 5280 feet | = | 1 mile |
| 4 inches | = | 1 hand |
| 9 inches | = | 1 span |
| 2½ feet | = | 1 pace |
| 16½ feet or 5½ yards | = | 1 rod |
| 1 knot or nautical mile | = | 6080.26 feet |
| | = | ¼ league |
| 7.92 inches | = | 1 link |
| 25 links | = | 1 rod |
| 100 links or 66 feet or 4 rods | = | 1 chain |
| 10 chains | = | 1 furlong |
| 8 furlongs | = | 1 mile |

Surface

| | | |
|-------------------|---|----------------|
| 144 square inches | = | 1 square foot |
| 9 square feet | = | 1 square yard |
| 30¼ square yards | = | 1 square rod |
| 160 square rods | = | 1 acre |
| 640 acres | = | 1 square mile |
| 625 square links | = | 1 square rod |
| 16 square rods | = | 1 square chain |
| 10 square chains | = | 1 acre |
| 640 acres | = | 1 square mile |
| 36 square miles | = | 1 township |

Volume

| | | |
|-------------------|---|--------------|
| 1728 cubic inches | = | 1 cubic foot |
| 27 cubic feet | = | 1 cubic yard |
| 128 cubic feet | = | 1 cord |
| 24½ cubic feet | = | 1 perch |

Troy Weight

| | |
|-----------------|------------------------|
| 24 grains (gr.) | = 1 pennyweight (dwt.) |
| 20 pennyweights | = 1 ounce (oz.) |
| 12 ounces | = 1 pound (lb.) |

Avoirdupois Weight

| | |
|---------------------------------|---------------------------|
| 16 drams (dr.) | = 1 ounce (oz.) |
| 16 ounces | = 1 pound (lb.) |
| 25 pounds | = 1 quarter (qr.) |
| 4 quarters | = 1 hundred weight (cwt.) |
| 20 hundred weight (2000 pounds) | = 1 ton (T.) |

Apothecaries' Weight

| | |
|-----------------|--------------------------------------|
| 20 grains (gr.) | = 1 scruple (sc. or \mathfrak{D}) |
| 3 scruples | = 1 dram (dr. or \mathfrak{S}) |
| 8 drams | = 1 ounce (oz. or $\mathfrak{℥}$) |
| 12 ounces | = 1 pound (lb) |

Dry Measure

| | |
|---------------|--------------------|
| 2 pints (pt.) | = 1 quart (qt.) |
| 8 quarts | = 1 peck (pk.) |
| 4 pecks | = 1 bushel (bu.) |
| 36 bushels | = 1 chaldron (ch.) |

Liquid Measure

| | |
|---------------|---------------------|
| 4 gills (gi.) | = 1 pint (pt.) |
| 2 pints | = 1 quart (qt.) |
| 4 quarts | = 1 gallon (gal.) |
| 31½ gallons | = 1 barrel (bar.) |
| 63 gallons | = 1 hogshead (hhd.) |

Apothecaries' Fluid Measure

| | |
|-----------------|------------------|
| 60 minims | = 1 fluid-drachm |
| 8 fluid-drachms | = 1 fluid-ounce |
| 16 fluid-ounces | = 1 pint |
| 8 pints | = 1 gallon |

Circular Measure

| | |
|--------------------------|-------------------|
| 60 seconds (") | = 1 minute (') |
| 60 minutes | = 1 degree (°) |
| 30 degrees | = 1 sign (s) |
| 12 signs, or 360 degrees | = 1 circle (cir.) |

English and Metric Conversion Tables

Length

| | |
|--------------|----------------------|
| 1 millimeter | = 39.370 mils |
| | = 0.039370 inch |
| 1 centimeter | = 0.39370 inch |
| | = 0.032808 foot |
| 1 inch | = 2.5400 centimeters |
| | = 0.083333 foot |
| 1 foot | = 30.480 centimeters |
| | = 0.30480 meter |
| 1 yard | = 91.440 centimeters |
| | = 0.91440 meter |
| 1 meter | = 39.370 inches |
| | = 3.2808 feet |
| | = 1.0936 yards |
| 1 kilometer | = 3280.8 feet |
| | = 1093.6 yards |
| | = 0.62137 mile |
| 1 mile | = 5280 feet |
| | = 1609.3 meters |
| | = 1.6093 kilometers |

Surface

| | |
|------------------|--------------------------------|
| 1 circular mil | = 0.78540 square mil |
| | = 0.00050671 square millimeter |
| 1 square mil | = 1.2732 circular mils |
| | = 0.00064516 square millimeter |
| | = 0.000001 square inch |
| 1 sq. millimeter | = 1973.5 circular mils |
| | = 1550.0 square mils |
| | = 0.0015500 square inch |
| 1 sq. centimeter | = 197,350 circular mils |
| | = 0.15500 square inch |
| 1 sq. inch | = 1,273,240 circular mils |
| | = 6.4516 square centimeters |
| 1 sq. foot | = 929.03 square centimeters |
| | = 144 square inches |

| | |
|-----------------|---|
| 1 sq. yard | = 1296 square inches = 9 square feet = 0.0083613 are = 0.00020661 acre |
| 1 sq. meter | = 1550.0 square inches = 10.764 square feet = 1.1960 square yards |
| 1 are | = 1076.4 square feet = 100 square meters |
| 1 acre | = 43,560 square feet = 4840 square yards = 4046.8 square meters = 0.40468 hectare = 0.0015625 square mile |
| 1 hectare | = 107,640 square feet = 100 ares = 2.4711 acres |
| 1 sq. kilometer | = 10,764,000 square feet = 1,196,000 square yards = 247.11 acres = 0.38610 square mile |
| 1 square mile | = 27,878,400 square feet = 3,097,600 square yards = 640 acres = 2.5900 square kilometers |

Volume

| | |
|------------------|---|
| 1 cu. centimeter | = 0.061024 cubic inch = 0.0021134 pint (liquid) = 0.0018162 pint (dry) |
| 1 cu. inch | = 16.387 cubic centimeters = 0.017317 quart (liquid) = 0.014881 quart (dry) = 0.016387 liter or cubic decimeter = 0.0043291 gallon = 0.00057870 cubic foot |
| 1 quart (liquid) | = 2 pints (liquid) = 946.33 cubic centimeters = 57.749 cubic inches = 0.94633 liter or cubic decimeter |

| | |
|---------------|-------------------------------------|
| 1 quart (dry) | = 2 pints (dry) |
| | = 1101.2 cubic centimeters |
| | = 67.199 cubic inches |
| | = 0.038889 cubic foot |
| 1 liter | = 1000 cubic centimeters |
| | = 61.024 cubic inches |
| | = 1.0567 quarts (dry) |
| | = 0.26418 gallon |
| 1 cubic foot | = 1728 cubic inches |
| | = 28.317 liters or cubic decimeters |
| | = 0.028317 cubic meter |
| 1 cubic yard | = 27 cubic feet |
| | = 0.76456 cubic meter |
| 1 gallon | = 3785.3 cubic centimeters |
| | = 230.99 cubic inches |
| | = 4 quarts (liquid) |
| | = 3.7853 liters |
| | = 0.13368 cubic foot |
| 1 cubic meter | = 35.315 cubic feet |
| | = 10 liters |
| | = 1.3080 cubic yards |
| | = 1 stere |

NOTE.—Pints, quarts, and gallons in this table refer to U. S. measures.

Weight

| | |
|---------------|-------------------------|
| 1 milligram | = 0.015432 grain |
| | = 0.001 gram |
| 1 grain * | = 64.799 milligrams |
| | = 0.0022857 ounce (av.) |
| 1 gram | = 15.432 grains |
| | = 0.035274 ounce (av.) |
| | = 0.0022046 pound (av.) |
| 1 ounce (av.) | = 437.50 grains |
| | = 28.350 grams |
| | = 0.062500 pound (av.) |

* The troy grain and the apothecaries' grain are of the same weight as the avoirdupois grain.

| | |
|-----------------|------------------------|
| 1 ounce (troy)* | = 31.103 grams |
| 1 pound (av.) | = 6999.97 grains |
| | = 453.59 grams |
| | = 16 ounces |
| | = 0.45359 kilogram |
| 1 kilogram | = 35.274 ounces (av.) |
| | = 2.2046 pounds (av.) |
| 1 ton (short) | = 2000 pounds (av.) |
| | = 907.18 kilograms |
| | = 0.89286 ton (long) |
| | = 0.90718 ton (metric) |
| 1 ton (metric) | = 2204.6 pounds |
| | = 1000 kilograms |
| | = 1.1023 ton (short) |
| | = 0.98425 ton (long) |
| 1 ton (long) | = 2240 pounds |
| | = 1.1200 ton (short) |
| | = 1.0160 ton (metric) |

Force

Equivalent of force given below are dependent on the value of g , the acceleration of gravity. The standard value of g adopted by the International Committee on Weights and Measures, is $g = 980.665$, corresponding to 45° latitude and sea-level.

| | |
|-----------|---------------------------|
| 1 dyne | = 0.01574 grain |
| | = 0.00102 gram |
| | = 0.00007233 poundal |
| | = 0.000002248 pound (av.) |
| 1 gram | = 980.6 dynes |
| | = 0.07093 poundal |
| 1 poundal | = 13,825 dynes |
| | = 0.03108 pound |
| | = 0.01410 kilogram |
| 1 pound | = 444,800 dynes |
| | = 32.17 poundals |

* The apothecaries' ounce is of the same weight as the troy ounce.

1 kilogram = 980600 dynes
 = 70.93 poundals

Storage of Water

1 acre-foot = 325,800 gallons
 = 43,560 cu. feet
 = 1613 cu. yards
 = 1233 cu. meters
 1 gallon = 0.00003069 acre-foot
 1 cu. foot = 0.00002298 acre-foot
 1 cu. yard = 0.00062 acre-foot

Temperature

1 degree Centigrade = $\frac{9}{5}$ (= 1.8) degree Fahrenheit
 1 degree Fahrenheit = $\frac{5}{9}$ (= 0.556) degree Centigrade
 temperature Fahr. = $t_f = \frac{9}{5} t_c + 32$
 temperature Cent. = $t_c = \frac{5}{9} (t_f - 32)$

Heat, Electric, and Mechanical Equivalents

Energy

1 erg = 1 dyne-cm.
 = 0.000001 joule
 = 0.0000007376 foot-pound
 1 gram-centimeter = 980.6 ergs
 = 0.00009806 joule
 = 0.00007233 foot-pound
 1 joule = 10,000,000 ergs
 = 0.7376 foot-pound
 = 0.2389 gram-calorie
 = 0.102 kilogram-meter
 = 0.0009480 B.t.u.
 = 0.0002778 watt-hour
 1 foot-pound = 13,560,000 ergs
 = 1.356 joules
 = 0.3239 gram-calorie
 = 0.1383 kilogram-meter
 = 0.001285 B.t.u.
 = 0.0003766 watt-hour
 = 0.0000005051 horsepower-hour

| | |
|--------------------|-----------------------------|
| 1 kilogram-meter | = 9.806 joules |
| | = 7.233 foot-pounds |
| | = 0.009296 B.t.u. |
| | = 0.002724 watt-hour |
| 1 B.t.u. | = 1055 joules |
| | = 778.1 foot-pounds |
| | = 252 gram-calories |
| | = 107.6 kilogram-meters |
| | = 0.2930 watt-hour |
| | = 0.0003930 horsepower-hour |
| 1 watt-hour | = 3600 joules |
| | = 2655.4 foot-pounds |
| | = 860 gram-calories |
| | = 3.413 B.t.u. |
| | = 0.001341 horsepower-hour |
| 1 kilogram-calorie | = 4186 joules |
| | = 3088 foot-pounds |
| | = 426.9 kilogram-meters |
| | = 1.163 watt-hours |
| 1 horsepower-hour | = 2,684,000 joules |
| | = 1,980,000 foot-pounds |
| | = 745.6 watt-hours |

Power

| | |
|------------------------------|------------------------------------|
| 1 erg per second | = 1 dyne-centimeter per second |
| | = 0.0000001 watt |
| 1 gram-centimeter per second | = 0.00009806 watt |
| 1 foot-pound per minute | = 0.02260 watt |
| | = 0.00003072 horsepower (metric) |
| | = 0.00003030 horsepower |
| 1 watt | = 44.26 foot-pounds per minute |
| | = 6.119 kilogram-meters per minute |
| 1 horsepower | = 33,000 foot-pounds per minute |
| | = 745.6 watts |
| | = 550 foot-pounds per second |
| | = 1.01387 horsepower (metric) |
| 1 horsepower (metric) | = 32,550 foot-pounds per minute |
| | = 735.5 watts |
| | = 75 kilogram-meters per second |
| | = 0.9863 horsepower |

| | |
|------------|-----------------------------------|
| 1 kilowatt | = 44,256.7 foot-pounds per minute |
| | = 1.3597 horsepower (metric) |
| | = 1.341 horsepower |

Electric Units

| | |
|------------|------------------|
| 1 abvolt | = 10^{-8} volt |
| 1 abampere | = 10 amperes |
| 1 abohm | = 10^{-9} ohm |

Pressure Equivalents

| | |
|-----------------------------|--|
| 1 atmosphere (standard) | = 29.9212 inches of mercury at 32° F. |
| | = 760 millimeters of mercury at 32° F. |
| | = 33.901 feet of water at 39.1° F. |
| | = 14.6969 pounds per sq. inch |
| | = 2116.35 pounds per sq. foot |
| 1 inch of mercury at 32° F. | = 0.491187 pound per sq. inch |
| | = 70.7310 pounds per sq. foot |
| | = 1.13299 feet of water at 39.1° F. |
| 1 foot of water at 39.1° F. | = 0.8826 inch of mercury at 32° F. |
| | = 62.425 pounds per sq. foot |
| | = 0.4335 pound per sq. inch |
| | = 0.0295 atmosphere |
| 1 pound on the sq. foot | = 0.016018 foot of water at 39.1° F. |
| 1 pound on the sq. inch | = 2.307 feet of water at 39.1° F. |

PRESSURE AND VOLUME CORRECTION, ETC.

Reduction of Barometer Readings to 0° C.

$$\text{corrected height } H_0 = H \left\{ 1 - \frac{(\beta - \alpha) t}{(1 + \beta t)} \right\}$$

H = observed height of barometer,

t = observed temperature of barometer in degrees Centigrade,

β = 0.0001818, the coefficient of cubical expansion of mercury,

α = coefficient of linear expansion of the material of the scale (0.0000085 for glass, 0.0000184 for brass).

Reduction of Gaseous Volumes to 0° C., and 1 Atmosphere Pressure

$$\text{corrected volume } v_0 = \left\{ \frac{v}{1 + 0.00367 t} \right\} \frac{p}{760}$$

v = observed volume,

t = observed temperature in degrees Centigrade,

p = pressure in millimeters of mercury.

Determination of Altitudes by the Barometer

For heights not exceeding 2000 feet, relative altitude is given by the approximate formula:

$$X \text{ (in feet)} = 52,500 \left\{ 1 + \frac{2(T + T_1)}{1000} \right\} \frac{H - H_1}{H + H_1}$$

X = vertical distance between the two stations,

T = Centigrade temperature at lower station,

T_1 = Centigrade temperature at upper station,

H = height of barometer at lower station reduced to 0° C.,

H_1 = height of barometer at upper station reduced to 0° C.

For any altitude,

$$X = 60,346 \left\{ 1 + 0.00256 \cos(2\theta) \right\} \left\{ 1 + 2 \frac{(T + T_1)}{1000} \right\} \log_{10} \frac{H}{H_1}$$

in which θ = latitude in degrees.

Velocity of Sound

The velocity of sound in gases is

$$V = \sqrt{\frac{\gamma P}{\rho}}$$

P = pressure,

ρ = density,

γ = ratio of specific heat at constant pressure to that at constant volume. (See Table, page 214.)

VELOCITY OF SOUND IN AIR AND WATER

| Substance | Temperature, Degrees C. | Velocity in meters per second | Velocity in feet per second |
|------------|----------------------------|-------------------------------------|-----------------------------------|
| Air..... | 0 | 331.7 | 1088 |
| Air..... | 20 | 344 | 1129 |
| Air..... | 100 | 386 | 1266 |
| Water..... | 13 | 1441 | 4728 |
| Water..... | 19 | 1461 | 4794 |
| Water..... | 31 | 1505 | 4938 |

Geodetic and Astronomical Data

Velocity of light = 186,330 miles per second

= 299,870 kilometers per second

Equatorial radius of the earth * = 3963.339 miles

= 6378.388 kilometers

Polar semi-diameter of the earth * = 3949.992 miles

= 6356.909 kilometers

Mean distance from the earth to the moon = 238,854 miles

= 384,393 kilometers

Mean distance from the earth to the sun = 92,900,000 miles

= 149,500,000 kilometers

* U. S. C. & G. Survey.

PHYSICAL AND CHEMICAL CONSTANTS

INTERNATIONAL ATOMIC WEIGHTS (1919)

| Element | Sym- bol | Atomic weight | Element | Sym- bol | Atomic weight |
|----------------|-------------|------------------|----------------|-------------|------------------|
| Aluminum..... | Al | 27.1 | Molybdenum.. | Mo | 96.0 |
| Antimony..... | Sb | 120.2 | Neodymium.. | Nd | 144.3 |
| Argon..... | A | 39.88 | Neon..... | Ne | 20.2 |
| Arsenic..... | As | 74.96 | Nickel..... | Ni | 58.68 |
| Barium..... | Ba | 137.37 | Niton..... | Nt | 222.4 |
| Bismuth..... | Bi | 208.0 | Nitrogen..... | N | 14.01 |
| Boron..... | B | 11.0 | Osmium..... | Os.. | 190.9 |
| Bromine..... | Br | 79.92 | Oxygen..... | O | 16.00 |
| Cadmium..... | Cd | 112.40 | Palladium..... | Pd | 106.7 |
| Cæsium..... | Cs | 132.81 | Phosphorus... | P | 31.04 |
| Calcium..... | Ca | 40.07 | Platinum..... | Pt | 195.2 |
| Carbon..... | C | 12.005 | Potassium.... | K | 39.10 |
| Cerium..... | Ce | 140.25 | Praseodymium | Pr | 140.9 |
| Chlorine..... | Cl | 35.46 | Radium..... | Ra | 226.0 |
| Chromium..... | Cr | 52.0 | Rhodium..... | Rh | 102.9 |
| Cobalt..... | Co | 58.97 | Rubidium.... | Rb | 85.45 |
| Columbium*.. | Cb | 93.1 | Ruthenium.... | Ru | 101.7 |
| Copper..... | Cu | 63.57 | Samarium.... | Sa | 150.4 |
| Dysprosium... | Dy | 162.5 | Scandium..... | Sc | 44.1 |
| Erbium..... | Er | 167.7 | Selenium..... | Se | 79.2 |
| Europium..... | Eu | 152.0 | Silicon..... | Si | 28.3 |
| Fluorine..... | F | 19.0 | Silver..... | Ag | 107.88 |
| Gadolinium.... | Gd | 157.3 | Sodium..... | Na | 23.00 |
| Gallium..... | Ga | 69.9 | Strontium.... | Sr | 87.63 |
| Germanium.... | Ge | 72.5 | Sulphur..... | S | 32.06 |
| Glucium †.... | Gl | 9.1 | Tantalum..... | Ta | 181.5 |
| Gold..... | Au | 197.2 | Tellurium.... | Te | 127.5 |
| Helium..... | He | 4.00 | Terbium..... | Tb | 159.2 |
| Holmium..... | Ho | 163.5 | Thallium..... | Tl | 204.0 |
| Hydrogen..... | H | 1.008 | Thorium..... | Th | 232.4 |
| Indium..... | In | 114.8 | Thulium..... | Tm | 168.5 |
| Iodine..... | I | 126.92 | Tin..... | Sn | 118.7 |
| Iridium..... | Ir | 193.1 | Titanium..... | Ti | 48.1 |
| Iron..... | Fe | 55.84 | Tungsten..... | W | 184.0 |
| Krypton..... | Kr | 82.92 | Uranium..... | U | 238.2 |
| Lanthanum.... | La | 139.0 | Vanadium.... | V | 51.0 |
| Lead..... | Pb | 207.20 | Xenon..... | Xe | 130.2 |
| Lithium..... | Li | 6.94 | Ytterbium.... | Yb | 173.5 |
| Lutecium..... | Lu | 175.0 | Yttrium..... | Yt | 88.7 |
| Magnesium.... | Mg | 24.32 | Zinc..... | Zn | 65.37 |
| Manganese.... | Mn | 54.93 | Zirconium.... | Zr | 90.6 |
| Mercury..... | Hg | 200.6 | | | |

* Columbium or Niobium (Nb).

† Glucium or Beryllium (Be).

WEIGHTS AND DENSITIES

| Element | Temperature, Degrees C.* | Density in grams per cu. centimeter † |
|------------------|-----------------------------|---|
| Aluminium | 20 | 2.70 |
| Antimony, pure | 20 | 6.618 |
| Compressed | 20 | 6.691 |
| Argon, liquid | -183 | 1.3845 |
| Arsenic, crys. | 14 | 5.73 |
| Barium | | 3.78 |
| Bismuth | 20 | 9.781 |
| Boron, crystal | | 2.535 |
| Amorphous | | 2.45 |
| Bromine, liquid | | 3.12 |
| Cadmium | 20 | 8.648 |
| Cæsium | 20 | 1.873 |
| Calcium | | 1.54 |
| Carbon, diamcnd. | | 3.52 |
| Graphite | | 2.25 |
| Cerium | | 7.02 |
| Chlorine, liquid | -33.6 | 1.507 |
| Chromium | 20 | 6.92 |
| Cobalt | 21 | 8.71 |
| Columbium | 15 | 8.4 |
| Copper | 20 | 8.89 |
| Erbium | | 4.77 |
| Fluorine, liquid | -200 | 1.14 |
| Gallium | 23 | 5.93 |
| Germanium | 20 | 5.46 |
| Glucinum | | 1.85 |
| Gold | | 9.33 |
| Helium, liquid | -269 | 0.15 |
| Hydrogen, liquid | -252 | 0.070 |
| Indium | | 7.28 |
| Iridium | 17 | 22.42 |
| Iodine | 20 | 4.940 |
| Iron, pure | | 7.86 |
| Wrought | | 7.8 to 7.9 |
| Krypton, liquid | -146 | 2.16 |
| Lanthanum | | 6.15 |
| Lead | 20 | 11.347 |
| Liquid | 325 | 10.645 |
| Lithium | 20 | 0.534 |
| Magnesium | | 1.741 |
| Manganese | | 7.42 |
| Mercury | 0 | 13.596 |

WEIGHT AND DENSITIES

| Element | Temperature, Degrees C.* | Density in grams per cu. centimeter † |
|----------------------------|-----------------------------|---|
| Mercury | 20 | 13.546 |
| Liquid | -38.8 | 13.690 |
| Solid | -38.8 | 14.193 |
| Molybdenum | | 9.01 |
| Neodymium | | 6.96 |
| Nickel | | 8.9 |
| Nitrogen, liquid | -195 | 0.810 |
| Osmium | | 22.5 |
| Oxygen | -184 | 1.14 |
| Palladium | | 12.16 |
| Phosphorus, red | | 2.20 |
| Yellow | | 1.83 |
| Platinum | 20 | 21.37 |
| Potassium | 20 | 0.870 |
| Praseodymium | | 6.475 |
| Rhodium | | 12.44 |
| Rubidium | 20 | 1.532 |
| Ruthenium | 0 | 12.06 |
| Samarium | | 7.7 to 7.8 |
| Selenium | | 4.3 to 4.8 |
| Silicon, crys. | 20 | 2.42 |
| Amorphous | 15 | 2.35 |
| Silver | 20 | 10.503 |
| Sodium | 20 | 0.9712 |
| Strontium | | 2.50 to 2.58 |
| Sulphur | | 2.0 to 2.1 |
| Tantalum | | 16.6 |
| Tellurium, amorphous | 20 | 6.02 |
| Thallium | | 11.86 |
| Thorium | 17 | 12.16 |
| Tin | | 7.29 |
| Titanium | 18 | 4.5 |
| Tungsten | | 18.6 to 19.1 |
| Uranium | 13 | 18.7 |
| Vanadium | | 5.69 |
| Xenon, liquid | -109 | 3.52 |
| Zinc | 20 | 7.13 |
| Zirconium | | 6.44 |

* Where temperature is not given, the value of density is for ordinary atmospheric temperatures.

† To reduce density in grams per cubic centimeter to pounds per cubic inch, multiply by 0.0361. To reduce density in grams per cubic centimeter to pounds per cubic foot, multiply by 62.4.

WEIGHTS AND DENSITIES (*Continued*)

| Miscellaneous substances | Density in grams per cubic centimeter | Pounds per cubic foot |
|--------------------------|---|--------------------------|
| Agate..... | 2.5 to 2.7 | 156 to 168 |
| Asbestos..... | 2.0 to 2.8 | 125 to 175 |
| Asphalt..... | 1.1 to 1.5 | 69 to 94 |
| Cement, set..... | 2.7 to 3.0 | 170 to 190 |
| Chalk..... | 1.9 to 2.8 | 118 to 175 |
| Clay..... | 1.8 to 2.6 | 122 to 162 |
| Coal, anthracite..... | 1.4 to 1.8 | 87 to 112 |
| Soft..... | 1.2 to 1.5 | 75 to 94 |
| Coke..... | 1.0 to 1.7 | 62 to 105 |
| Dolomite..... | 2.84 | 177 |
| Ebonite..... | 1.15 | 72 |
| Feldspar..... | 2.55 to 2.75 | 159 to 172 |
| Flint..... | 2.63 | 164 |
| Fluorite..... | 3.18 | 198 |
| Glass, common..... | 2.4 to 2.8 | 150 to 175 |
| Granite..... | 2.64 to 2.76 | 165 to 172 |
| Graphite..... | 2.30 to 2.72 | 144 to 170 |
| Hornblende..... | 3.0 | 187 |
| Ice..... | 0.917 | 57.2 |
| Ivory..... | 1.83 to 1.92 | 114 to 120 |
| Lime, mortar..... | 1.65 to 1.78 | 103 to 111 |
| Slaked..... | 1.3 to 1.4 | 81 to 87 |
| Limestone..... | 2.68 to 2.76 | 167 to 171 |
| Magnetite..... | 4.9 to 5.2 | 306 to 324 |
| Malachite..... | 3.7 to 4.1 | 231 to 256 |
| Marble..... | 2.6 to 2.84 | 160 to 177 |
| Mica..... | 2.6 to 3.2 | 165 to 200 |
| Paraffin..... | 0.87 to 0.91 | 54 to 57 |
| Pyrite..... | 4.95 to 5.1 | 309 to 318 |
| Quartz..... | 2.65 | 165 |
| Quartzite..... | 2.73 | 170 |
| Sandstone..... | 2.14 to 2.36 | 134 to 147 |
| Slate..... | 2.6 to 3.3 | 162 to 205 |

WEIGHTS AND DENSITIES (*Continued*)

| Woods * | Density in grams per cubic centimeter | Weight in pounds per cubic foot |
|-------------------|---|---------------------------------------|
| Ash..... | 0.65 to 0.85 | 40 to 53 |
| Beech..... | 0.70 to 0.90 | 43 to 56 |
| Cedar..... | 0.49 to 0.57 | 30 to 35 |
| Cork..... | 0.22 to 0.26 | 14 to 16 |
| Elm..... | 0.54 to 0.60 | 34 to 37 |
| Fir..... | 0.48 to 0.70 | 30 to 44 |
| Lignum-vitæ..... | 1.17 to 1.33 | 73 to 83 |
| Mahogany..... | 0.85 | 53 |
| Maple..... | 0.62 to 0.75 | 39 to 47 |
| Oak..... | 0.60 to 0.90 | 37 to 56 |
| Pine, yellow..... | 0.37 to 0.60 | 23 to 37 |
| Pine, white..... | 0.35 to 0.50 | 22 to 31 |
| Poplar..... | 0.35 to 0.50 | 22 to 31 |
| Spruce..... | 0.48 to 0.70 | 30 to 44 |
| Walnut..... | 0.64 to 0.70 | 40 to 43 |

* Seasoned and of average dryness.

Values for gases given below are for 0° Cent. (32° Fahr.) and a pressure of one atmosphere.

| Gases | Density relative to air | Weight in grams per liter | Weight in pounds per cubic foot |
|--|-------------------------------|---------------------------------|---------------------------------------|
| Acetylene, C ₂ H ₂ | 0.920 | 1.1620 | 0.07254 |
| Air..... | 1.000 | 1.2928 | 0.08071 |
| Ammonia, NH ₃ | 0.597 | 0.7706 | 0.04811 |
| Carbon monoxide, CO..... | 0.9672 | 1.2506 | 0.07807 |
| Carbon dioxide, CO ₂ | 1.5291 | 1.9768 | 0.12341 |
| Ethane, C ₂ H ₆ | 1.0494 | 1.3567 | 0.08470 |
| Hydrochloric acid, HCl..... | 1.2684 | 1.6398 | 0.10237 |
| Hydrogen, H ₂ | 0.0696 | 0.09004 | 0.005621 |
| Hydrogen sulphide, H ₂ S..... | 1.1895 | 1.5230 | 0.09508 |
| Methane, CH ₄ | 0.5576 | 0.7160 | 0.04470 |
| Nitrous oxide, N ₂ O..... | 1.5298 | 1.9777 | 0.12347 |
| Nitric oxide, NO..... | 1.0367 | 1.3402 | 0.08367 |
| Nitrogen, N ₂ | 0.9673 | 1.2514 | 0.07812 |
| Oxygen, O ₂ | 1.1053 | 1.4292 | 0.08922 |
| Sulphur dioxide, SO ₂ | 2.2639 | 2.9266 | 0.18271 |

WEIGHTS AND DENSITIES (*Continued*)

| Liquids | Temperature, Degrees C. | Density in grams per cubic centimeter | Pounds per cubic foot |
|-----------------------------|-------------------------|---------------------------------------|-----------------------|
| Acid, hydrochloric..... | | 1.20 | 74.8 |
| Acid, nitric..... | | 1.22 | 76.0 |
| Acid, sulphuric..... | | 1.84 | 116.5 |
| Alcohol, ethyl..... | 0 | 0.807 | 50.4 |
| Alcohol, methyl..... | 0 | 0.810 | 50.5 |
| Carbolic acid..... | 15 | 0.95 to 0.965 | 59.2 to 60.2 |
| Carbon disulphide..... | 0 | 1.293 | 80.6 |
| Gasoline..... | | 0.66 to 0.69 | 41 to 43 |
| Glycerine..... | 0 | 1.26 | 78.6 |
| Naphtha..... | 15 | 0.665 | 41.5 |
| Oil, linseed..... | 15 | 0.942 | 58.8 |
| Oil, olive..... | 15 | 0.918 | 57.3 |
| Petroleum..... | 0 | 0.878 | 54.8 |
| Turpentine..... | 16 | 0.873 | 54.2 |
| Water (freezing-point)..... | 0 | 0.99987 | 62.417 |
| (maximum density)..... | 4 | 1.0000 | 62.425 |
| (standard 62° F.)..... | 16.7 | 0.99886 | 62.354 |
| | 20 | 0.99823 | 62.315 |
| | 100 | 0.9584 | 59.70 |
| Water, sea (62° F.)..... | 16.7 | 1.0260 | 63.976 |

MELTING AND BOILING POINTS OF ELEMENTS

| Element | Melting point | | Boiling point at atmospheric pressure | |
|------------------|---------------|--------------|---------------------------------------|------------|
| | Degrees C. | Degrees F. | Degrees C. | Degrees F. |
| Aluminium | 657 | 1215 | 1800 | 3272 |
| Antimony | 630 | 1166 | 1440 | 2624 |
| Argon | -188 | -306 | -186 | -303 |
| Arsenic | (volatilizes) | | (sublimes) | |
| | | | 450 | 842 |
| Barium | 850 | 1562 | | |
| Bismuth | 269 | 516 | 1420 | 2590 |
| Boron | 2000 to 2500 | 3630 to 4530 | (sublimes) | |
| | | | 3500 | 6330 |
| Bromine | -7.3 | 18.9 | 63 | 145.5 |
| Cadmium | 321 | 610 | 778 | 1432 |
| Cæsium | 26.4 | 79.5 | 670 | 1238 |
| Calcium | 780 | 1436 | | |
| Carbon | 4000 | 7230 | | |
| Cerium | 623 | 1153 | | |
| Chlorine | -102 | -151.6 | -33.6 | -28.5 |
| Chromium | 1520 | 2768 | 2200 | 3992 |
| Cobalt | 1480 | 2696 | | |
| Columbium | 1950 | 3542 | | |
| Copper | 1083 | 1982 | 2310 | 4190 |
| Fluorine | -223 | -369 | -187 | -305 |
| Gallium | 30.2 | 86.4 | | |
| Glucinum | 1430 | 2606 | | |
| Gold | 1063 | 1945 | 2530 | 4586 |
| Helium | below -272 | below -458 | -268.8 | -451 |
| Hydrogen | -259 | -434 | -252.7 | -423 |
| Indium | 155 | 311 | 1000 | 1830 |
| Iodine | 113 | 235 | 184.4 | 364 |
| Iridium | 2290 | 4150 | 2550 | 4610 |
| Iron | 1530 | 2786 | 2450 | 4442 |
| Krypton | -169 | -272 | -151.7 | -241.1 |
| Lanthanum | 810 | 1490 | | |
| Lead | 327 | 621 | 1525 | 2779 |
| Lithium | 186 | 367 | 1400 | 2552 |
| Magnesium | 633 | 1171 | 1120 | 2048 |
| Manganese | 1260 | 2320 | 1900 | 3452 |
| Mercury | -38.87 | -37.98 | 356.7 | 674 |
| Molybdenum | 2450 | 4440 | 3200 | 5790 |
| Nickel | 1452 | 2646 | 2330 | 4226 |

MELTING AND BOILING POINTS OF ELEMENTS
 (Continued)

| Element | Melting point | | Boiling point at atmospheric pressure | |
|-------------------|---------------|------------|---------------------------------------|------------|
| | Degrees C. | Degrees F. | Degrees C. | Degrees F. |
| Nitrogen..... | -210.5 | -347 | -195.7 | -320 |
| Osmium..... | 2700 | 4890 | | |
| Oxygen..... | -219 | -362 | -182.9 | -297 |
| Palladium..... | 1549 | 2820 | 2540 | 4600 |
| Phosphorus..... | 44.1 | 111.4 | 287 | 549 |
| Platinum..... | 1755 | 3190 | 2450 | 4440 |
| Potassium..... | 62.5 | 144.5 | 758 | 1396 |
| Praseodymium..... | 940 | 1724 | | |
| Radium..... | 700 | 1290 | | |
| Rhodium..... | 1907 | 3465 | 2500 | 4530 |
| Rubidium..... | 38.5 | 111.3 | 696 | 1285 |
| Ruthenium..... | 1900 | 3450 | 2520 | 4570 |
| Samarium..... | 1350 | 2460 | | |
| Selenium..... | 217 | 423 | 690 | 1274 |
| Silicon..... | 1420 | 2588 | 3500 | 6330 |
| Silver..... | 961 | 1762 | 1955 | 3551 |
| Sodium..... | 97.0 | 206.6 | 750 | 1380 |
| Strontium..... | 900 | 1650 | | |
| Sulphur..... | 115 | 239 | 444.6 | 832.3 |
| Tantalum..... | 2910 | 5270 | | |
| Tellurium..... | 450 | 840 | 1390 | 3530 |
| Thallium..... | 301 | 574 | 1280 | 2790 |
| Thorium..... | 1690 | 3070 | | |
| Tin..... | 232 | 449.6 | 2270 | 4118 |
| Titanium..... | 1795 | 3440 | | |
| Tungsten..... | 3500 | 6330 | 3700 | 6690 |
| Vanadium..... | 1720 | 3130 | | |
| Xenon..... | -140 | -220 | -109.1 | -164.4 |
| Zinc..... | 418 | 784 | 918 | 1684 |
| Zirconium..... | 2300 | 4170 | | |

SPECIFIC HEATS

| Element | Temperature, Degrees C. | Specific heat |
|-------------------------|----------------------------|------------------|
| Aluminium..... | 16 to 100 | 0.2122 |
| Antimony..... | 17 to 92 | 0.0508 |
| Arsenic, cryst..... | 0 to 100 | 0.0861 |
| Arsenic, amorphous..... | 0 to 100 | 0.0822 |
| Barium..... | -185 to 20 | 0.068 |
| Beryllium..... | 0 to 100 | 0.425 |
| Bismuth..... | 20 to 100 | 0.0302 |
| Bismuth, fluid..... | 280 to 380 | 0.0363 |
| Boron..... | 0 to 100 | 0.307 |
| Bromine, solid..... | -78 to -20 | 0.0843 |
| Bromine, fluid..... | 13 to 45 | 0.107 |
| Cadmium..... | 18 to 99 | 0.055 |
| Cæsium..... | 0 to 26 | 0.0482 |
| Calcium..... | 0 to 100 | 0.149 |
| Carbon, graphite..... | 11 | 0.160 |
| Carbon, diamond..... | 11 | 0.113 |
| Cerium..... | 0 to 100 | 0.0448 |
| Chlorine, liquid..... | 0 to 24 | 0.2262 |
| Chromium..... | 0 | 0.1039 |
| Chromium..... | 100 | 0.1121 |
| Cobalt..... | 15 to 100 | 0.1030 |
| Copper..... | 20 to 100 | 0.0936 |
| Gallium, solid..... | 12 to 23 | 0.079 |
| Gallium, liquid..... | 30 to 113 | 0.080 |
| Germanium..... | 0 to 100 | 0.0737 |
| Gold..... | 0 to 100 | 0.0316 |
| Indium..... | 0 to 100 | 0.0570 |
| Iodine..... | 9 to 98 | 0.0541 |
| Iridium..... | 18 to 100 | 0.0323 |
| Iron, cast..... | 20 to 100 | 0.1189 |
| Iron, wrought..... | 15 to 100 | 0.1152 |
| Iron, wrought..... | 0 to 1100 | 0.153 |
| Iron, hard-drawn..... | 20 to 100 | 0.1146 |
| Lanthanum..... | 0 to 100 | 0.0448 |
| Lead..... | 20 to 100 | 0.0305 |
| Lead..... | 300 | 0.0338 |
| Lithium..... | 0 to 100 | 1.093 |
| Magnesium..... | 20 to 100 | 0.2492 |
| Manganese..... | 20 to 100 | 0.1211 |
| Mercury..... | 20 | 0.0333 |
| Molybdenum..... | 20 to 100 | 0.0647 |
| Nickel..... | 18 to 100 | 0.109 |
| Osmium..... | 19 to 98 | 0.0311 |
| Palladium..... | 0 to 100 | 0.0592 |

SPECIFIC HEATS (*Continued*)

| Element | Temperature, Degrees C. | Specific heat |
|--------------------------|----------------------------|------------------|
| Phosphorus, red..... | 0 to 51 | 0.1829 |
| Phosphorus, yellow..... | 13 to 36 | 0.202 |
| Platinum..... | 0 to 100 | 0.0323 |
| Potassium..... | -78 to 23 | 0.166 |
| Rhodium..... | 10 to 97 | 0.0580 |
| Ruthenium..... | 0 to 100 | 0.0611 |
| Selenium, cryst..... | 22 to 62 | 0.084 |
| Selenium, amorphous..... | 18 to 38 | 0.095 |
| Silicon..... | 57.1 | 0.1833 |
| Silver..... | 0 to 100 | 0.0559 |
| Sodium..... | 10 | 0.297 |
| Sulphur, rhombic..... | 0 to 54 | 0.1728 |
| Sulphur, monoclinic..... | 0 to 52 | 0.1809 |
| Sulphur, liquid..... | 119 to 147 | 0.235 |
| Tantalum..... | 58 | 0.036 |
| Tellurium, cryst..... | 15 to 100 | 0.0483 |
| Thallium..... | 20 to 100 | 0.0326 |
| Thorium..... | 0 to 100 | 0.0276 |
| Tin..... | 19 to 29 | 0.0552 |
| Tin, molten..... | 250 | 0.05799 |
| Titanium..... | 0 to 100 | 0.1125 |
| Tungsten..... | 0 to 100 | 0.0336 |
| Uranium..... | 0 to 98 | 0.028 |
| Vanadium..... | 0 to 100 | 0.1153 |
| Zinc..... | 0 to 100 | 0.0935 |
| Zinc..... | 300 | 0.1040 |
| Zirconium..... | 0 to 100 | 0.0660 |

| Liquids | Temperature Degrees C. | Specific heat |
|--------------------------------|---------------------------|------------------|
| Alcohol, ethyl..... | 40 | 0.648 |
| Alcohol, methyl..... | 15 to 20 | 0.601 |
| Benzene..... | 10 | 0.340 |
| Benzene..... | 40 | 0.423 |
| Brine (density 1.2)..... | -20 | 0.69 |
| Glycerine..... | 15 to 50 | 0.576 |
| Oil, olive..... | 7 | 0.47 |
| Petroleum..... | 21 to 58 | 0.511 |
| Sea-water (density 1.024)..... | 17.5 | 0.938 |
| Turpentine..... | 18 | 0.42 |
| Water..... | 0 | 1.0094 |
| Water..... | 20 | 1.0000 |
| Water..... | 100 | 1.0074 |

SPECIFIC HEATS (*Continued*)

| Gases | Specific heat at constant pressure | | Ratio, $\frac{c_p}{c_v}$, of the specific heat at constant pressure to that of constant volume | |
|------------------------|------------------------------------|---------------|---|--------------------------|
| | Temperature range, Degrees C. | Specific heat | Temperature range, Degrees C. | Ratio, $\frac{c_p}{c_v}$ |
| Air..... | 0 to 200 | 0.2375 | 0 | 1.402 |
| Air..... | | | 500 | 1.399 |
| Alcohol, ethyl..... | 108 to 220 | 0.4534 | 53 | 1.133 |
| Alcohol..... | | | 100 | 1.134 |
| Ammonia..... | 24 to 216 | 0.5125 | 0 | 1.317 |
| Ammonia..... | | | 100 | 1.277 |
| Benzene..... | 34 to 115 | 0.2990 | 60 | 1.403 |
| Carbon monoxide..... | 26 to 198 | 0.2426 | 0 | 1.403 |
| Carbon monoxide..... | | | 100 | 1.395 |
| Carbon dioxide..... | 11 to 214 | 0.2169 | 4 to 11 | 1.300 |
| Carbon dioxide..... | | | 500 | 1.260 |
| Carbon disulphide..... | 86 to 190 | 0.1596 | 3 to 67 | 1.205 |
| Ethylene..... | | 0.4040 | | 1.264 |
| Hydrogen..... | 12 to 198 | 3.4090 | 4 to 16 | 1.408 |
| Methane..... | 18 to 208 | 0.5929 | 11 to 30 | 1.316 |
| Nitrogen..... | 0 to 200 | 0.2438 | | 1.410 |
| Oxygen..... | 13 to 207 | 0.2175 | 5 to 14 | 1.398 |
| Oxygen..... | 20 to 440 | 0.2240 | | |

SPECIFIC HEATS (*Continued*)

| Miscellaneous substances | Temperature, Degrees C. | Specific heat |
|--------------------------|----------------------------|------------------|
| Asbestos..... | 20 to 98 | 0.195 |
| Brass..... | 14 to 98 | 0.0862 |
| Charcoal..... | 0 to 224 | 0.238 |
| Glass, crown..... | 10 to 50 | 0.161 |
| Glass, flint..... | 10 to 50 | 0.117 |
| Granite..... | 12 to 100 | 0.192 |
| Ice..... | -21 to -1 | 0.502 |
| India rubber..... | 15 to 100 | 0.27 to 0.48 |
| Limestone..... | 15 to 100 | 0.216 |
| Marble..... | 0 to 100 | 0.21 |
| Masonry..... | | 0.20 |
| Paraffin wax, solid..... | 0 to 20 | 0.694 |
| Paraffin wax, fluid..... | 60 to 63 | 0.712 |
| Porcelain..... | 15 to 1000 | 0.255 |
| Quartz..... | 20 to 98 | 0.191 |
| Sandstone..... | | 0.22 |
| Vulcanite..... | 20 to 100 | 0.331 |

NOTE.—The specific heat of a material is the number of British Thermal Units necessary to raise the temperature of 1 pound of the material 1° Fahrenheit.

Coefficients of Linear Expansion of Solids

The length of a solid at any temperature is $l_t = l_0(1 + \alpha t)$, l_0 being the known length at some given temperature, t the variation of temperature in degrees, and α the coefficient of linear expansion of the material. This formula holds approximately when the temperature interval is not large. The coefficient of **surface expansion** equals 2α ; the coefficient of **cubical expansion** equals 3α .

COEFFICIENTS OF LINEAR EXPANSION (α)

The values given for α are the mean coefficients of expansion between 0° and 100° C., when some other temperature is not specified.

| Elements | Temperature | Coefficient of linear expansion | |
|----------------------|-------------|---------------------------------|------------------|
| | | For 1° C. | For 1° F. |
| Aluminium..... | | 0.00002220 | 0.00001233 |
| Antimony..... | | 0.00001056 | 0.00000587 |
| Arsenic..... | 40 | 0.00000559 | 0.00000311 |
| Bismuth..... | | 0.00001316 | 0.00000731 |
| Cadmium..... | | 0.00003159 | 0.00001755 |
| Carbon, diamond.... | 40 | 0.00000118 | 0.00000066 |
| Carbon, anthracite.. | 40 | 0.00002078 | 0.00001154 |
| Carbon, graphite.... | 40 | 0.00000786 | 0.00000437 |
| Cobalt..... | 40 | 0.00001236 | 0.00000687 |
| Copper..... | | 0.00001666 | 0.00000926 |
| Gold..... | | 0.00001470 | 0.00000817 |
| Indium..... | 40 | 0.00004170 | 0.00002317 |
| Iron, cast..... | 40 | 0.00001061 | 0.00000589 |
| Iron, annealed..... | | 0.00001089 | 0.00000605 |
| Lead..... | | 0.00002709 | 0.00001505 |
| Magnesium..... | 40 | 0.00002694 | 0.00001497 |
| Nickel..... | 40 | 0.00001279 | 0.00000710 |
| Osmium..... | 40 | 0.00000657 | 0.00000365 |
| Palladium..... | 40 | 0.00001176 | 0.00000653 |
| Phosphorus..... | 0 to 40 | 0.00012530 | 0.00006961 |
| Platinum..... | 40 | 0.00000899 | 0.00000499 |
| Potassium..... | 0 to 50 | 0.00008300 | 0.00004611 |
| Rhodium..... | 40 | 0.00000850 | 0.00000472 |
| Ruthenium..... | 40 | 0.00000963 | 0.00000535 |
| Selenium..... | 40 | 0.00003680 | 0.00002044 |
| Silicon..... | 40 | 0.00000763 | 0.00000424 |
| Silver..... | 40 | 0.00001921 | 0.00001067 |
| Sulphur..... | | 0.00011800 | 0.00006556 |
| Tellurium..... | | 0.00003687 | 0.00002048 |
| Thallium..... | 40 | 0.00003021 | 0.00001678 |
| Tin..... | | 0.00002296 | 0.00001276 |
| Zinc..... | | 0.00002976 | 0.00001653 |

COEFFICIENTS OF LINEAR EXPANSION (α)
 (Continued)

| Miscellaneous substances | Temperature | Coefficient of linear expansion | |
|--------------------------|-------------|---------------------------------|------------|
| | | For 1° C. | For 1° F. |
| Brass, cast..... | | 0.00001875 | 0.00001042 |
| Brass, wire..... | | 0.00001930 | 0.00001072 |
| Bronze..... | 16.6 to 100 | 0.00001844 | 0.00001024 |
| Ebonite..... | 25 to 35 | 0.0000842 | 0.0000468 |
| German silver..... | | 0.00001836 | 0.00001020 |
| Glass, crown..... | | 0.00000897 | 0.00000498 |
| Glass, flint..... | 50 to 60 | 0.00000788 | 0.00000530 |
| Glass, plate..... | | 0.00000891 | 0.00000495 |
| Glass, tube..... | | 0.00000833 | 0.00000463 |
| Gutta percha..... | 20 | 0.0001983 | 0.0001102 |
| Ice..... | -20 to -1 | 0.000051 | 0.000028 |
| Marble..... | 15 to 100 | 0.0000117 | 0.0000065 |
| Paraffin wax..... | 0 to 16 | 0.00010662 | 0.00005923 |
| Paraffin wax..... | 16 to 38 | 0.00013030 | 0.00007239 |
| Porcelain..... | 20 to 790 | 0.00000413 | 0.00000229 |
| Quartz: | | | |
| Parallel to axis... | 0 to 80 | 0.00000797 | 0.00000443 |
| Perpend. to axis.. | 0 to 80 | 0.00001337 | 0.00000743 |

| Woods * | Coefficient of linear expansion | |
|-------------------|---------------------------------|------------|
| | For 1° C. | For 1° F. |
| (1) Along grain: | | |
| Beech..... | 0.00000257 | 0.00000143 |
| Chestnut..... | 0.00000649 | 0.00000361 |
| Elm..... | 0.00000565 | 0.00000314 |
| Mahogany..... | 0.00000361 | 0.00000201 |
| Maple..... | 0.00000638 | 0.00000347 |
| Oak..... | 0.00000492 | 0.00000273 |
| Pine..... | 0.00000541 | 0.00000301 |
| Walnut..... | 0.00000658 | 0.00000366 |
| (2) Across grain: | | |
| Beech..... | 0.0000614 | 0.0000363 |
| Chestnut..... | 0.0000325 | 0.0000181 |
| Elm..... | 0.0000443 | 0.0000246 |
| Mahogany..... | 0.0000404 | 0.0000224 |
| Maple..... | 0.0000484 | 0.0000269 |
| Oak..... | 0.0000544 | 0.0000302 |
| Pine..... | 0.0000341 | 0.0000189 |
| Walnut..... | 0.0000484 | 0.0000269 |

* For temperature range 2° to 34° Cent.

PROPERTIES OF SATURATED STEAM

Tables condensed with permission from G. A. Goodenough's "Properties of Steam and Ammonia," published by Messrs. John Wiley and Sons.

| Absolute pressure in inches of mercury | Temp. Fahr. | Volume of one pound in cu. ft., v' | Heat content in B.t.u. | | Latent heat in B.t.u. | | Entropy | | |
|--|-------------|--------------------------------------|------------------------|-----------------|-----------------------|----------------------|------------------------|---|------------------------|
| | | | of liquid, q | of vapor, v'' | Total, L or r | Internal, I or p | of liquid, h or s' | of vaporization, $\frac{L}{T}$ or $\frac{r}{T}$ | of vapor, N or s'' |
| 1 | 79.06 | 652 | 47.11 | 1095.0 | 1047.9 | 968.7 | 0.0915 | 1.9455 | 2.0370 |
| 2 | 101.17 | 338.9 | 69.16 | 1105.1 | 1036.0 | 974.3 | 0.1316 | 1.8474 | 1.9790 |
| 3 | 115.06 | 231.4 | 83.04 | 1111.4 | 1028.3 | 965.2 | 0.1561 | 1.7893 | 1.9454 |
| 4 | 125.44 | 176.5 | 93.37 | 1115.9 | 1022.5 | 958.3 | 0.1739 | 1.7478 | 1.9217 |
| 5 | 133.78 | 143.2 | 101.68 | 1119.6 | 1017.9 | 952.8 | 0.1880 | 1.7154 | 1.9034 |
| 6 | 140.80 | 120.7 | 108.69 | 1122.6 | 1013.9 | 948.1 | 0.1998 | 1.6888 | 1.8886 |
| 7 | 146.88 | 110.4 | 114.8 | 1125.2 | 1010.5 | 944.0 | 0.2098 | 1.6661 | 1.8760 |
| 8 | 152.26 | 92.1 | 120.2 | 1127.5 | 1007.4 | 940.4 | 0.2187 | 1.6464 | 1.8651 |
| 9 | 157.10 | 82.5 | 125.0 | 1129.6 | 1004.6 | 937.1 | 0.2265 | 1.6290 | 1.8556 |
| 10 | 161.50 | 74.8 | 129.4 | 1131.4 | 1002.1 | 934.1 | 0.2336 | 1.6134 | 1.8470 |
| 11 | 165.55 | 68.4 | 133.4 | 1133.1 | 999.7 | 931.3 | 0.2401 | 1.5992 | 1.8393 |
| 12 | 169.30 | 63.0 | 137.2 | 1134.7 | 997.5 | 928.8 | 0.2461 | 1.5862 | 1.8323 |
| 13 | 172.79 | 58.5 | 140.7 | 1136.1 | 995.5 | 926.4 | 0.2516 | 1.5742 | 1.8258 |
| 14 | 176.06 | 54.6 | 143.9 | 1137.5 | 993.6 | 924.1 | 0.2568 | 1.5630 | 1.8198 |
| 15 | 179.14 | 51.14 | 147.0 | 1138.8 | 991.7 | 922.0 | 0.2617 | 1.5526 | 1.8143 |
| 16 | 182.06 | 48.14 | 149.9 | 1140.0 | 990.0 | 920.0 | 0.2662 | 1.5429 | 1.8091 |
| 17 | 184.83 | 45.49 | 152.7 | 1141.1 | 988.3 | 918.1 | 0.2705 | 1.5337 | 1.8042 |
| 18 | 187.46 | 43.12 | 155.4 | 1142.1 | 986.7 | 916.2 | 0.2746 | 1.5250 | 1.7996 |
| 19 | 189.97 | 40.99 | 157.9 | 1143.1 | 985.2 | 914.4 | 0.2785 | 1.5168 | 1.7953 |
| 20 | 192.38 | 39.08 | 160.3 | 1144.1 | 983.8 | 912.7 | 0.2822 | 1.5089 | 1.7912 |

PROPERTIES OF SATURATED STEAM (Continued)

| Absolute pressure in inches of mercury | Temp. Fahr. | Volume of one pound in cu. ft., v' | Heat content in B.t.u. | | Latent heat in B.t.u. | | Entropy | | |
|--|-------------|--------------------------------------|------------------------|-----------------|-----------------------|----------------------|----------------------------------|--|----------------------------------|
| | | | of liquid, h' | of vapor, h'' | Total, L or τ | Internal, J or p | of liquid, $\frac{h}{T}$ or s' | of vaporization, $\frac{\tau}{T}$ or $\frac{r}{T}$ | of vapor, $\frac{h}{T}$ or s'' |
| 21 | 194.68 | 37.34 | 162.6 | 1145.0 | 982.4 | 911.1 | 0.2858 | 1.5015 | 1.7873 |
| 22 | 196.89 | 35.75 | 164.8 | 1145.9 | 981.1 | 909.6 | 0.2892 | 1.4944 | 1.7855 |
| 23 | 199.03 | 34.29 | 167.0 | 1146.7 | 979.8 | 908.1 | 0.2924 | 1.4876 | 1.7800 |
| 24 | 201.09 | 32.95 | 169.0 | 1147.5 | 978.5 | 906.6 | 0.2955 | 1.4810 | 1.7766 |
| 25 | 203.08 | 31.71 | 170.1 | 1148.3 | 977.3 | 905.2 | 0.2986 | 1.4747 | 1.7733 |
| 26 | 205.00 | 30.57 | 173.0 | 1149.1 | 976.1 | 903.8 | 0.3015 | 1.4687 | 1.7702 |
| 27 | 206.87 | 29.51 | 174.8 | 1149.8 | 974.9 | 902.5 | 0.3043 | 1.4629 | 1.7671 |
| 28 | 208.67 | 28.53 | 176.6 | 1150.5 | 973.8 | 901.2 | 0.3070 | 1.4572 | 1.7642 |
| 29 | 210.43 | 27.61 | 178.4 | 1151.2 | 972.7 | 900.0 | 0.3096 | 1.4518 | 1.7614 |
| in pounds per sq. inch | | | | | | | | | |
| 14.7° | 212.0 | 26.81 | 180.0 | 1151.7 | 971.7 | 898.8 | 0.3120 | 1.4469 | 1.7589 |
| 15 | 213.0 | 26.30 | 181.0 | 1152.2 | 971.2 | 898.1 | 0.3135 | 1.4438 | 1.7573 |
| 16 | 216.3 | 24.76 | 184.3 | 1153.4 | 969.1 | 895.8 | 0.3184 | 1.4337 | 1.7521 |
| 17 | 219.4 | 23.40 | 187.5 | 1154.6 | 967.1 | 893.5 | 0.3230 | 1.4242 | 1.7473 |
| 18 | 222.4 | 22.18 | 190.5 | 1155.7 | 965.2 | 891.4 | 0.3274 | 1.4153 | 1.7427 |
| 19 | 225.2 | 21.09 | 193.3 | 1156.7 | 963.4 | 889.3 | 0.3316 | 1.4068 | 1.7384 |
| 20 | 228.0 | 20.10 | 196.0 | 1157.7 | 961.7 | 887.3 | 0.3356 | 1.3987 | 1.7343 |
| 22 | 233.1 | 18.38 | 201.2 | 1159.6 | 958.4 | 883.6 | 0.3430 | 1.3837 | 1.7267 |
| 24 | 237.8 | 16.95 | 206.0 | 1161.3 | 955.3 | 880.1 | 0.3499 | 1.3698 | 1.7197 |
| 26 | 242.2 | 15.73 | 210.4 | 1162.8 | 952.4 | 876.8 | 0.3563 | 1.3570 | 1.7133 |
| 28 | 246.4 | 14.67 | 214.6 | 1164.3 | 949.7 | 873.7 | 0.3622 | 1.3452 | 1.7074 |

PROPERTIES OF SATURATED STEAM (Continued)

| Absolute pressure in pounds per sq. in. | Temp. Fahr. | Volume of one pound in cu. ft., v' | Heat content in B.t.u. | | Latent heat in B.t.u. | | Entropy | | |
|---|-------------|--------------------------------------|------------------------|-----------------|-----------------------|-------------------------|--------------------------|---|------------------------|
| | | | of liquid, f' | of vapor, f'' | Total, L or r | Internal, I or ρ | of liquid, π or s' | of vaporization, $\frac{r}{T}$ or $\frac{r}{T}$ | of vapor, N or s'' |
| 30 | 250.3 | 13.76 | 218.6 | 1165.7 | 947.1 | 870.7 | 0.3679 | 1.3340 | 1.7019 |
| 32 | 254.0 | 12.95 | 222.4 | 1166.9 | 944.6 | 867.9 | 0.3731 | 1.3236 | 1.6967 |
| 34 | 257.6 | 12.24 | 225.9 | 1168.1 | 942.2 | 865.2 | 0.3781 | 1.3137 | 1.6918 |
| 36 | 260.9 | 11.60 | 229.4 | 1169.2 | 939.9 | 862.7 | 0.3829 | 1.3044 | 1.6873 |
| 38 | 264.2 | 11.03 | 232.6 | 1170.3 | 937.7 | 860.2 | 0.3874 | 1.2956 | 1.6830 |
| 40 | 267.2 | 10.51 | 235.8 | 1171.3 | 935.5 | 857.8 | 0.3917 | 1.2871 | 1.6788 |
| 42 | 270.2 | 10.04 | 238.8 | 1172.2 | 933.5 | 855.5 | 0.3958 | 1.2791 | 1.6749 |
| 44 | 273.0 | 9.61 | 241.7 | 1173.2 | 931.5 | 853.3 | 0.3998 | 1.2714 | 1.6712 |
| 46 | 275.8 | 9.22 | 244.5 | 1174.0 | 929.6 | 851.2 | 0.4036 | 1.2640 | 1.6676 |
| 48 | 278.4 | 8.86 | 247.2 | 1174.8 | 927.7 | 849.1 | 0.4072 | 1.2570 | 1.6642 |
| 50 | 281.0 | 8.53 | 249.8 | 1175.6 | 925.9 | 847.1 | 0.4108 | 1.2501 | 1.6609 |
| 52 | 283.5 | 8.22 | 252.3 | 1176.4 | 924.1 | 845.1 | 0.4142 | 1.2436 | 1.6577 |
| 54 | 285.9 | 7.93 | 254.7 | 1177.1 | 922.4 | 843.2 | 0.4174 | 1.2373 | 1.6547 |
| 56 | 288.2 | 7.67 | 257.1 | 1177.8 | 920.7 | 841.4 | 0.4206 | 1.2311 | 1.6517 |
| 58 | 290.5 | 7.42 | 259.5 | 1178.5 | 919.0 | 839.5 | 0.4237 | 1.2252 | 1.6489 |
| 60 | 292.7 | 7.18 | 261.7 | 1179.1 | 917.4 | 837.8 | 0.4267 | 1.2195 | 1.6462 |
| 62 | 294.9 | 6.97 | 263.9 | 1179.7 | 915.8 | 836.0 | 0.4296 | 1.2139 | 1.6435 |
| 64 | 296.9 | 6.76 | 266.1 | 1180.3 | 914.3 | 834.3 | 0.4324 | 1.2085 | 1.6409 |
| 66 | 299.0 | 6.57 | 268.2 | 1180.9 | 912.7 | 832.7 | 0.4352 | 1.2032 | 1.6384 |
| 68 | 301.0 | 6.39 | 270.2 | 1181.5 | 911.2 | 831.1 | 0.4379 | 1.1981 | 1.6360 |

PROPERTIES OF SATURATED STEAM (Continued)

| Absolute pressure in pounds per sq. in. | Temp. Fahr. | Volume of one pound in cu. ft., v' | Heat content in B.t.u. | | Latent heat in B.t.u. | | Entropy | | |
|---|-------------|--------------------------------------|--------------------------|-------------------------|-----------------------|-------------------------|--------------------------|---|------------------------|
| | | | of liquid, $\frac{p}{p}$ | of vapor, $\frac{v}{v}$ | Total, L or r | Internal, I or ρ | of liquid, π or s' | of vaporization, $\frac{r}{T}$ or $\frac{r}{T}$ | of vapor, N or s'' |
| 70 | 302.9 | 6.22 | 272.2 | 1182.0 | 909.8 | 829.5 | 0.4405 | 1.1931 | 1.6336 |
| 72 | 304.8 | 6.05 | 274.2 | 1182.5 | 908.3 | 827.9 | 0.4431 | 1.1883 | 1.6313 |
| 74 | 306.7 | 5.90 | 276.1 | 1183.0 | 906.9 | 826.4 | 0.4456 | 1.1835 | 1.6291 |
| 76 | 308.5 | 5.75 | 278.0 | 1183.5 | 905.5 | 824.9 | 0.4480 | 1.1789 | 1.6269 |
| 78 | 310.3 | 5.61 | 279.8 | 1184.0 | 904.2 | 823.4 | 0.4504 | 1.1744 | 1.6246 |
| 80 | 312.0 | 5.48 | 281.6 | 1184.4 | 902.8 | 821.9 | 0.4527 | 1.1700 | 1.6227 |
| 82 | 313.7 | 5.35 | 283.4 | 1184.9 | 901.5 | 820.5 | 0.4550 | 1.1657 | 1.6207 |
| 84 | 315.4 | 5.23 | 285.1 | 1185.3 | 900.2 | 819.1 | 0.4572 | 1.1615 | 1.6187 |
| 86 | 317.1 | 5.12 | 286.8 | 1185.7 | 898.9 | 817.7 | 0.4594 | 1.1574 | 1.6168 |
| 88 | 318.7 | 5.01 | 288.5 | 1186.1 | 897.7 | 816.3 | 0.4615 | 1.1534 | 1.6149 |
| 90 | 320.3 | 4.905 | 290.1 | 1186.5 | 896.4 | 815.0 | 0.4636 | 1.1495 | 1.6131 |
| 92 | 321.8 | 4.805 | 291.7 | 1186.9 | 895.2 | 813.7 | 0.4657 | 1.1456 | 1.6113 |
| 94 | 323.3 | 4.709 | 293.3 | 1187.3 | 894.0 | 812.4 | 0.4677 | 1.1419 | 1.6096 |
| 96 | 324.8 | 4.617 | 294.8 | 1187.7 | 892.8 | 811.1 | 0.4697 | 1.1381 | 1.6079 |
| 98 | 326.3 | 4.528 | 296.4 | 1188.0 | 891.6 | 809.8 | 0.4717 | 1.1345 | 1.6062 |
| 100 | 327.8 | 4.442 | 297.9 | 1188.4 | 890.5 | 808.6 | 0.4736 | 1.1309 | 1.6045 |
| 102 | 329.2 | 4.359 | 299.4 | 1188.7 | 889.3 | 807.4 | 0.4755 | 1.1274 | 1.6028 |
| 104 | 330.7 | 4.279 | 300.9 | 1189.0 | 888.2 | 806.1 | 0.4773 | 1.1239 | 1.6012 |
| 106 | 332.0 | 4.202 | 302.3 | 1189.4 | 887.1 | 804.9 | 0.4791 | 1.1205 | 1.5996 |
| 108 | 333.4 | 4.128 | 303.7 | 1189.7 | 885.9 | 803.8 | 0.4809 | 1.1172 | 1.5981 |

PROPERTIES OF SATURATED STEAM (Continued)

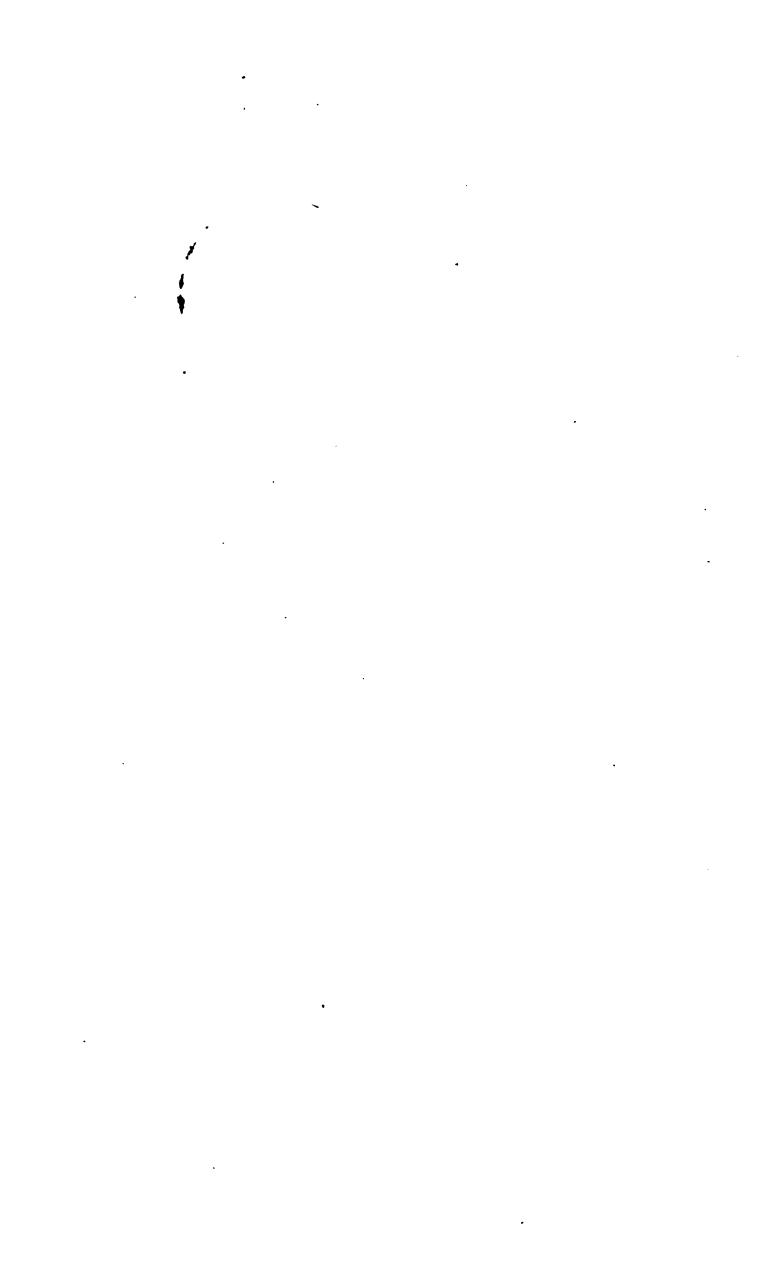
| Absolute pressure in pounds per sq. in. | Temp. Fahr. | Volume of one pound in cu. ft., v' | Heat content in B.t.u. | | Latent heat in B.t.u. | | Entropy | | |
|---|-------------|--------------------------------------|--------------------------|--------------------------|-----------------------|-------------------------|----------------------------------|---|----------------------------------|
| | | | of liquid, $\frac{L}{v}$ | of vapor, $\frac{L}{v'}$ | Total, L or τ | Internal, I or ρ | of liquid, $\frac{L}{v}$ or s' | of vaporization, $\frac{L}{v} - \frac{L}{v'}$ | of vapor, $\frac{L}{v}$ or s'' |
| 110 | 334.8 | 4.057 | 305.1 | 1190.0 | 884.8 | 802.6 | 0.4827 | 1.1138 | 1.5965 |
| 112 | 336.1 | 3.968 | 306.5 | 1190.3 | 883.7 | 801.4 | 0.4844 | 1.1106 | 1.5950 |
| 114 | 337.4 | 3.921 | 307.9 | 1190.6 | 882.7 | 800.3 | 0.4861 | 1.1074 | 1.5935 |
| 116 | 338.7 | 3.857 | 309.2 | 1190.8 | 881.6 | 799.2 | 0.4878 | 1.1043 | 1.5921 |
| 118 | 340.0 | 3.795 | 310.6 | 1191.1 | 880.6 | 798.0 | 0.4895 | 1.1012 | 1.5907 |
| 120 | 341.3 | 3.735 | 311.9 | 1191.4 | 879.5 | 796.9 | 0.4911 | 1.0982 | 1.5893 |
| 122 | 342.5 | 3.676 | 313.2 | 1191.6 | 878.5 | 795.8 | 0.4927 | 1.0952 | 1.5879 |
| 124 | 343.7 | 3.620 | 314.4 | 1191.9 | 877.5 | 794.8 | 0.4943 | 1.0922 | 1.5865 |
| 126 | 345.0 | 3.566 | 315.7 | 1192.1 | 876.4 | 793.7 | 0.4958 | 1.0894 | 1.5852 |
| 128 | 346.2 | 3.513 | 316.9 | 1192.4 | 875.4 | 792.6 | 0.4974 | 1.0865 | 1.5838 |
| 130 | 347.4 | 3.461 | 318.2 | 1192.6 | 874.4 | 791.6 | 0.4989 | 1.0836 | 1.5825 |
| 132 | 348.5 | 3.412 | 319.4 | 1192.9 | 873.5 | 790.5 | 0.5004 | 1.0808 | 1.5812 |
| 134 | 349.7 | 3.363 | 320.6 | 1193.1 | 872.5 | 789.5 | 0.5019 | 1.0781 | 1.5800 |
| 136 | 350.8 | 3.316 | 321.8 | 1193.3 | 871.5 | 788.5 | 0.5033 | 1.0754 | 1.5787 |
| 138 | 352.0 | 3.270 | 323.0 | 1193.5 | 870.5 | 787.4 | 0.5048 | 1.0727 | 1.5775 |
| 140 | 353.1 | 3.226 | 324.2 | 1193.7 | 869.6 | 786.4 | 0.5062 | 1.0700 | 1.5762 |
| 142 | 354.2 | 3.182 | 325.3 | 1193.9 | 868.6 | 785.4 | 0.5076 | 1.0674 | 1.5750 |
| 144 | 355.3 | 3.140 | 326.5 | 1194.1 | 867.7 | 784.5 | 0.5090 | 1.0648 | 1.5738 |
| 146 | 356.3 | 3.099 | 327.6 | 1194.3 | 866.8 | 783.5 | 0.5104 | 1.0623 | 1.5727 |
| 148 | 357.4 | 3.059 | 328.7 | 1194.5 | 865.8 | 782.5 | 0.5117 | 1.0598 | 1.5715 |

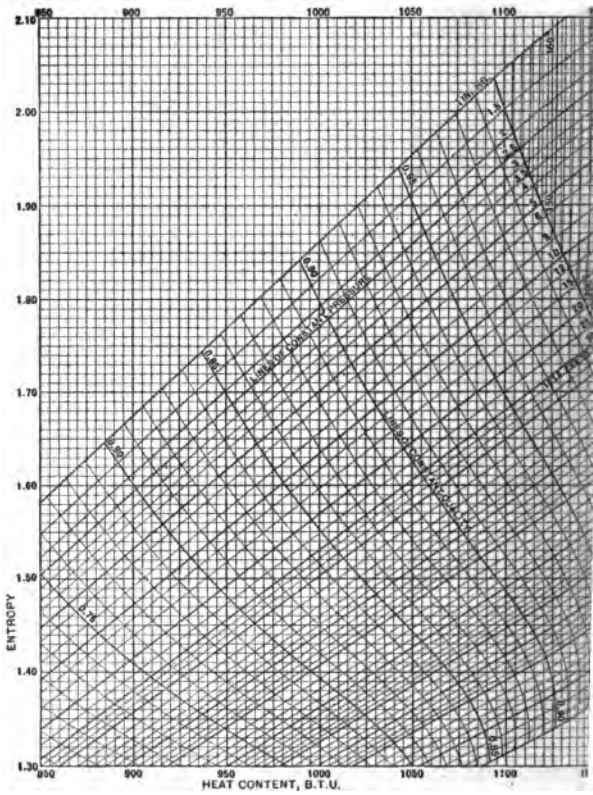
PROPERTIES OF SATURATED STEAM (Continued)

| Absolute pressure in pounds per sq. in. | Temp. Fahr. | Volume of one pound in cu. ft., v' | Heat content in B.t.u. | | Latent heat in B.t.u. | | Entropy | | |
|---|-------------|--------------------------------------|------------------------|----------------|-----------------------|-------------------------|-----------------------------|---|------------------------|
| | | | of liquid, p | of vapor, v' | Total, L or r | Internal, I or ρ | of liquid, κ or s' | of vaporization, $\frac{r}{T}$ or $\frac{r}{T}$ | of vapor, N or s'' |
| 150 | 358.5 | 3.020 | 329.8 | 1194.7 | 864.9 | 781.6 | 0.5131 | 1.0573 | 1.5704 |
| 152 | 359.5 | 2.982 | 330.9 | 1194.9 | 864.0 | 780.6 | 0.5144 | 1.0548 | 1.5692 |
| 154 | 360.5 | 2.945 | 332.0 | 1195.1 | 863.1 | 779.7 | 0.5157 | 1.0524 | 1.5681 |
| 156 | 361.6 | 2.909 | 333.1 | 1195.3 | 862.3 | 778.7 | 0.5170 | 1.0500 | 1.5670 |
| 158 | 362.6 | 2.874 | 334.1 | 1195.5 | 861.4 | 777.8 | 0.5183 | 1.0476 | 1.5659 |
| 160 | 363.6 | 2.839 | 335.2 | 1195.7 | 860.5 | 776.9 | 0.5196 | 1.0453 | 1.5649 |
| 162 | 364.6 | 2.806 | 336.2 | 1195.8 | 859.6 | 776.0 | 0.5209 | 1.0429 | 1.5638 |
| 164 | 365.6 | 2.773 | 337.3 | 1196.0 | 858.7 | 775.1 | 0.5221 | 1.0406 | 1.5627 |
| 166 | 366.5 | 2.741 | 338.3 | 1196.2 | 857.9 | 774.2 | 0.5233 | 1.0384 | 1.5617 |
| 168 | 367.5 | 2.710 | 339.3 | 1196.3 | 857.0 | 773.3 | 0.5245 | 1.0361 | 1.5607 |
| 170 | 368.5 | 2.679 | 340.3 | 1196.5 | 856.2 | 772.4 | 0.5258 | 1.0339 | 1.5597 |
| 172 | 369.4 | 2.649 | 341.3 | 1196.6 | 855.3 | 771.5 | 0.5270 | 1.0317 | 1.5587 |
| 174 | 370.4 | 2.620 | 342.3 | 1196.8 | 854.5 | 770.6 | 0.5281 | 1.0295 | 1.5577 |
| 176 | 371.3 | 2.591 | 343.3 | 1196.9 | 853.6 | 769.8 | 0.5293 | 1.0274 | 1.5567 |
| 178 | 372.2 | 2.563 | 344.3 | 1197.1 | 852.8 | 768.9 | 0.5305 | 1.0252 | 1.5557 |
| 180 | 373.1 | 2.536 | 345.2 | 1197.2 | 852.0 | 768.0 | 0.5316 | 1.0231 | 1.5547 |
| 182 | 374.0 | 2.509 | 346.2 | 1197.4 | 851.2 | 767.2 | 0.5328 | 1.0210 | 1.5538 |
| 184 | 374.9 | 2.483 | 347.1 | 1197.5 | 850.4 | 766.4 | 0.5339 | 1.0189 | 1.5528 |
| 186 | 375.8 | 2.457 | 348.1 | 1197.6 | 849.5 | 765.5 | 0.5350 | 1.0169 | 1.5519 |
| 188 | 376.7 | 2.432 | 349.0 | 1197.8 | 848.7 | 764.7 | 0.5361 | 1.0148 | 1.5509 |

PROPERTIES OF SATURATED STEAM (Continued)

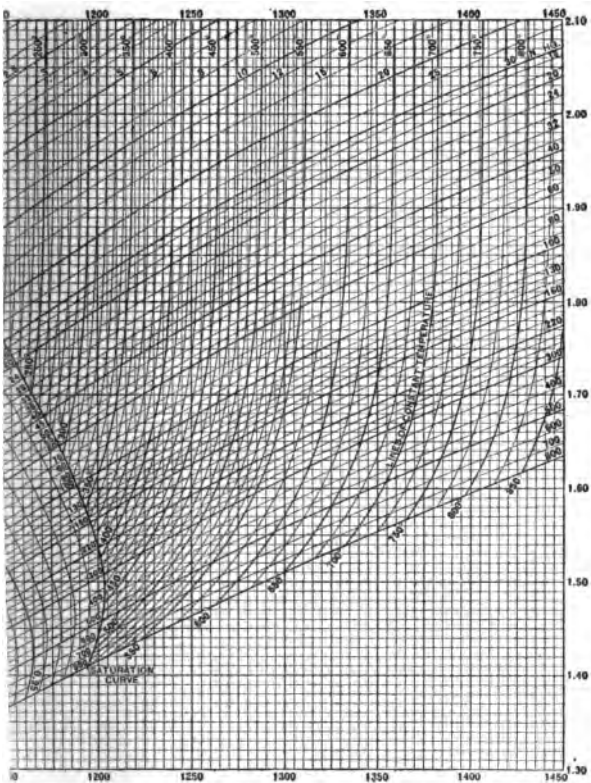
| Absolute pressure in pounds per sq. in. | Temp. Fahr. | Volume of one pound in cu. ft., v'' | Heat content in B.t.u. | | Latent heat in B.t.u. | | Entropy | | |
|---|-------------|---------------------------------------|------------------------|---------------|-----------------------|----------------------|-------------------------|---|------------------------|
| | | | of liquid, f | of vapor, g | Total, L or l | Internal, I or p | of liquid, $\#$ or s' | of vaporization, $\frac{L}{T}$ or $\frac{l}{T}$ | of vapor, N or s'' |
| 190 | 377.6 | 2.406 | 350.0 | 1197.9 | 847.9 | 763.9 | 0.5372 | 1.0128 | 1.5500 |
| 192 | 378.5 | 2.383 | 350.9 | 1198.0 | 847.1 | 763.0 | 0.5383 | 1.0108 | 1.5491 |
| 194 | 379.3 | 2.360 | 351.8 | 1198.1 | 846.3 | 762.2 | 0.5394 | 1.0089 | 1.5482 |
| 196 | 380.2 | 2.337 | 352.7 | 1198.2 | 845.6 | 761.4 | 0.5404 | 1.0069 | 1.5473 |
| 198 | 381.0 | 2.314 | 353.6 | 1198.4 | 844.8 | 760.6 | 0.5415 | 1.0049 | 1.5464 |
| 200 | 381.9 | 2.292 | 354.5 | 1198.5 | 844.0 | 759.8 | 0.5426 | 1.0030 | 1.5456 |
| 205 | 383.9 | 2.238 | 356.7 | 1198.7 | 842.1 | 757.8 | 0.5451 | 0.9983 | 1.5434 |
| 210 | 386.0 | 2.186 | 358.8 | 1199.0 | 840.2 | 755.9 | 0.5477 | 0.9956 | 1.5413 |
| 215 | 388.0 | 2.137 | 361.0 | 1199.2 | 838.3 | 754.0 | 0.5502 | 0.9930 | 1.5392 |
| 220 | 390.0 | 2.090 | 363.0 | 1199.5 | 836.5 | 752.1 | 0.5526 | 0.9846 | 1.5372 |
| 225 | 391.9 | 2.045 | 365.1 | 1199.7 | 834.6 | 750.2 | 0.5550 | 0.9802 | 1.5352 |
| 230 | 393.8 | 2.002 | 367.1 | 1199.9 | 832.8 | 748.3 | 0.5573 | 0.9760 | 1.5333 |
| 235 | 395.6 | 1.961 | 369.1 | 1200.1 | 831.0 | 746.5 | 0.5597 | 0.9717 | 1.5314 |
| 240 | 397.5 | 1.921 | 371.0 | 1200.3 | 829.3 | 744.7 | 0.5619 | 0.9676 | 1.5295 |
| 245 | 399.3 | 1.883 | 373.0 | 1200.5 | 827.5 | 742.9 | 0.5641 | 0.9635 | 1.5276 |
| 250 | 401.1 | 1.846 | 374.9 | 1200.6 | 825.8 | 741.2 | 0.5663 | 0.9595 | 1.5258 |
| 255 | 402.9 | 1.811 | 376.7 | 1200.8 | 824.1 | 739.5 | 0.5685 | 0.9556 | 1.5241 |
| 260 | 404.5 | 1.777 | 378.6 | 1201.0 | 822.4 | 737.7 | 0.5706 | 0.9517 | 1.5223 |
| 265 | 406.2 | 1.745 | 380.4 | 1201.1 | 820.7 | 736.0 | 0.5727 | 0.9479 | 1.5206 |
| 270 | 407.9 | 1.713 | 382.2 | 1201.2 | 819.1 | 734.4 | 0.5747 | 0.9442 | 1.5189 |





MOLLIER'S

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EAM CHART

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PROPERTIES OF SATURATED STEAM (Continued)

| Absolute pressure in pounds per sq. in. | Temp. Fabr. | Volume of one pound in cu. ft., v' | Heat content in B.t.u. | | Latent heat in B.t.u. | | Entropy | | |
|---|-------------|--------------------------------------|------------------------|------------------|-----------------------|----------------------|--------------------------|--|--------|
| | | | of vapor, p_v | of liquid, p_l | Total, L or T | Internal, I or P | of liquid, π or s' | of vaporization, L or T or $\frac{r}{T}$ | |
| | | | | | | | | N or s'' | |
| 275 | 409.6 | 1.683 | 383.9 | 1201.4 | 817.4 | 732.7 | 0.5767 | 0.9405 | 1.5172 |
| 280 | 411.2 | 1.654 | 385.7 | 1201.5 | 815.8 | 731.1 | 0.5787 | 0.9369 | 1.5156 |
| 285 | 412.8 | 1.625 | 387.4 | 1201.6 | 814.2 | 729.5 | 0.5806 | 0.9333 | 1.5139 |
| 290 | 414.4 | 1.598 | 389.1 | 1201.7 | 812.6 | 727.9 | 0.5826 | 0.9298 | 1.5123 |
| 295 | 415.9 | 1.571 | 390.8 | 1201.8 | 811.0 | 726.3 | 0.5845 | 0.9263 | 1.5108 |
| 300 | 417.5 | 1.545 | 392.4 | 1201.9 | 809.4 | 724.7 | 0.5863 | 0.9229 | 1.5092 |
| 310 | 420.5 | 1.496 | 395.7 | 1202.0 | 806.4 | 721.6 | 0.5900 | 0.9162 | 1.5062 |
| 320 | 423.4 | 1.450 | 398.9 | 1202.2 | 803.3 | 718.5 | 0.5935 | 0.9097 | 1.5032 |
| 330 | 426.3 | 1.407 | 402.0 | 1202.3 | 800.3 | 715.6 | 0.5970 | 0.9034 | 1.5004 |
| 340 | 429.1 | 1.366 | 405.0 | 1202.4 | 797.4 | 712.6 | 0.6004 | 0.8972 | 1.4976 |
| 350 | 431.9 | 1.327 | 408.0 | 1202.5 | 794.5 | 709.7 | 0.6036 | 0.8912 | 1.4949 |
| 360 | 434.6 | 1.291 | 410.9 | 1202.5 | 791.6 | 706.9 | 0.6068 | 0.8854 | 1.4922 |
| 370 | 437.2 | 1.256 | 413.7 | 1202.6 | 788.8 | 704.1 | 0.6100 | 0.8796 | 1.4896 |
| 380 | 439.8 | 1.223 | 416.5 | 1202.6 | 786.1 | 701.4 | 0.6130 | 0.8741 | 1.4871 |
| 390 | 442.3 | 1.192 | 419.3 | 1202.6 | 783.3 | 698.7 | 0.6161 | 0.8686 | 1.4847 |
| 400 | 444.8 | 1.162 | 422.0 | 1202.5 | 780.6 | 695.9 | 0.6190 | 0.8631 | 1.4821 |
| 450 | 456.5 | 1.033 | 434.8 | 1202.2 | 767.4 | 683.1 | 0.6329 | 0.8377 | 1.4706 |
| 500 | 467.2 | 0.928 | 446.6 | 1201.7 | 755.0 | 670.9 | 0.6455 | 0.8146 | 1.4601 |
| 600 | 486.5 | 0.770 | 468.0 | 1199.8 | 731.8 | 648.5 | 0.6679 | 0.7735 | 1.4414 |
| 700 | 503.4 | 0.656 | 487.1 | 1197.4 | 710.3 | 627.9 | 0.6874 | 0.7376 | 1.4250 |

TABLES

CIRCUMFERENCES AND AREAS OF CIRCLES

| Diameter | Circumference | Area | Diameter | Circumference | Area |
|----------|---------------|---------|----------|---------------|---------|
| 1 | 3.1416 | 0.7854 | 26 | 81.681 | 530.93 |
| 2 | 6.2832 | 3.1416 | 27 | 84.823 | 572.56 |
| 3 | 9.4248 | 7.0686 | 28 | 87.965 | 615.75 |
| 4 | 12.5664 | 12.5664 | 29 | 91.106 | 660.52 |
| 5 | 15.7080 | 19.635 | 30 | 94.248 | 706.86 |
| 6 | 18.850 | 28.274 | 31 | 97.389 | 754.77 |
| 7 | 21.991 | 38.485 | 32 | 100.53 | 804.25 |
| 8 | 25.133 | 50.266 | 33 | 103.67 | 855.30 |
| 9 | 28.274 | 63.617 | 34 | 106.81 | 907.92 |
| 10 | 31.416 | 78.540 | 35 | 109.96 | 962.11 |
| 11 | 34.558 | 95.033 | 36 | 113.10 | 1017.88 |
| 12 | 37.699 | 113.10 | 37 | 116.24 | 1075.21 |
| 13 | 40.841 | 132.73 | 38 | 119.38 | 1134.11 |
| 14 | 43.982 | 153.94 | 39 | 122.52 | 1194.59 |
| 15 | 47.124 | 176.71 | 40 | 125.66 | 1256.64 |
| 16 | 50.265 | 201.06 | 41 | 128.81 | 1320.25 |
| 17 | 53.407 | 226.98 | 42 | 131.95 | 1385.44 |
| 18 | 56.549 | 254.47 | 43 | 135.09 | 1452.20 |
| 19 | 59.690 | 283.53 | 44 | 138.23 | 1520.53 |
| 20 | 62.832 | 314.16 | 45 | 141.37 | 1590.43 |
| 21 | 65.973 | 346.36 | 46 | 144.51 | 1661.90 |
| 22 | 69.115 | 380.13 | 47 | 147.65 | 1734.94 |
| 23 | 72.257 | 415.48 | 48 | 150.80 | 1809.56 |
| 24 | 75.398 | 452.39 | 49 | 153.94 | 1885.74 |
| 25 | 78.540 | 490.87 | 50 | 157.08 | 1963.50 |

Note. — The surface of a sphere of given diameter may be found directly from the above table, since it is equal to the area of a circle of twice the diameter of the sphere.

CIRCUMFERENCES AND AREAS OF CIRCLES
(Continued)

| Diameter | Circumference | Area | Diameter | Circumference | Area |
|----------|---------------|---------|----------|---------------|---------|
| 51 | 160.22 | 2042.82 | 76 | 238.76 | 4536.46 |
| 52 | 163.36 | 2123.72 | 77 | 241.90 | 4656.63 |
| 53 | 166.50 | 2206.18 | 78 | 245.04 | 4778.36 |
| 54 | 169.65 | 2290.22 | 79 | 248.19 | 4901.67 |
| 55 | 172.79 | 2375.83 | 80 | 251.33 | 5026.55 |
| 56 | 175.93 | 2463.01 | 81 | 254.47 | 5153.00 |
| 57 | 179.07 | 2551.76 | 82 | 257.61 | 5281.02 |
| 58 | 182.21 | 2642.08 | 83 | 260.75 | 5410.61 |
| 59 | 185.35 | 2733.97 | 84 | 263.89 | 5541.77 |
| 60 | 188.50 | 2827.43 | 85 | 267.04 | 5674.50 |
| 61 | 191.64 | 2922.47 | 86 | 270.18 | 5808.80 |
| 62 | 194.78 | 3019.07 | 87 | 273.32 | 5944.68 |
| 63 | 197.92 | 3117.25 | 88 | 276.46 | 6082.12 |
| 64 | 201.06 | 3216.99 | 89 | 279.60 | 6221.14 |
| 65 | 204.20 | 3318.31 | 90 | 282.74 | 6361.73 |
| 66 | 207.34 | 3421.19 | 91 | 285.88 | 6503.88 |
| 67 | 210.49 | 3525.65 | 92 | 289.03 | 6647.61 |
| 68 | 213.63 | 3631.68 | 93 | 292.17 | 6792.91 |
| 69 | 216.77 | 3739.28 | 94 | 295.31 | 6939.78 |
| 70 | 219.91 | 3848.45 | 95 | 298.45 | 7088.22 |
| 71 | 223.05 | 3959.19 | 96 | 301.59 | 7238.23 |
| 72 | 226.19 | 4071.50 | 97 | 304.73 | 7389.81 |
| 73 | 229.34 | 4185.39 | 98 | 307.88 | 7542.96 |
| 74 | 232.48 | 4300.84 | 99 | 311.02 | 7697.69 |
| 75 | 235.62 | 4417.86 | 100 | 314.16 | 7853.98 |

POWERS, ROOTS, AND RECIPROCAL

| Number | Square | Cube | Square root | Cube root | Reciprocal |
|--------|--------|--------|-------------|-----------|------------|
| 1 | 1 | 1 | 1.000000 | 1.000000 | 1.0000000 |
| 2 | 4 | 8 | 1.414214 | 1.259921 | .5000000 |
| 3 | 9 | 27 | 1.732051 | 1.442250 | .3333333 |
| 4 | 16 | 64 | 2.000000 | 1.587401 | .2500000 |
| 5 | 25 | 125 | 2.236068 | 1.709976 | .2000000 |
| 6 | 36 | 216 | 2.449490 | 1.817121 | .1666667 |
| 7 | 49 | 343 | 2.645751 | 1.912931 | .1428571 |
| 8 | 64 | 512 | 2.828427 | 2.000000 | .1250000 |
| 9 | 81 | 729 | 3.000000 | 2.080084 | .1111111 |
| 10 | 100 | 1000 | 3.162278 | 2.154435 | .1000000 |
| 11 | 121 | 1331 | 3.316625 | 2.223980 | .0909091 |
| 12 | 144 | 1728 | 3.464102 | 2.289429 | .0833333 |
| 13 | 169 | 2197 | 3.605551 | 2.351335 | .0769231 |
| 14 | 196 | 2744 | 3.741657 | 2.410142 | .0714286 |
| 15 | 225 | 3375 | 3.872983 | 2.466212 | .0666667 |
| 16 | 256 | 4096 | 4.000000 | 2.519842 | .0625000 |
| 17 | 289 | 4913 | 4.123106 | 2.571282 | .0588235 |
| 18 | 324 | 5832 | 4.242641 | 2.620741 | .0555556 |
| 19 | 361 | 6859 | 4.358899 | 2.668402 | .0526316 |
| 20 | 400 | 8000 | 4.472136 | 2.714418 | .0500000 |
| 21 | 441 | 9261 | 4.582576 | 2.758924 | .0476190 |
| 22 | 484 | 10,648 | 4.690416 | 2.802039 | .0454545 |
| 23 | 529 | 12,167 | 4.795832 | 2.843867 | .0434783 |
| 24 | 576 | 13,824 | 4.898980 | 2.884499 | .0416667 |
| 25 | 625 | 15,625 | 5.000000 | 2.924018 | .0400000 |
| 26 | 676 | 17,576 | 5.099020 | 2.962496 | .0384615 |
| 27 | 729 | 19,683 | 5.196152 | 3.000000 | .0370370 |
| 28 | 784 | 21,952 | 5.291503 | 3.036589 | .0357143 |
| 29 | 841 | 24,389 | 5.385165 | 3.072317 | .0344828 |
| 30 | 900 | 27,000 | 5.477226 | 3.107233 | .0333333 |
| 31 | 961 | 29,791 | 5.567764 | 3.141381 | .0322581 |
| 32 | 1024 | 32,768 | 5.656854 | 3.174802 | .0312500 |
| 33 | 1089 | 35,937 | 5.744563 | 3.207534 | .0303030 |
| 34 | 1156 | 39,304 | 5.830952 | 3.239612 | .0294118 |
| 35 | 1225 | 42,875 | 5.916080 | 3.271066 | .0285714 |
| 36 | 1296 | 46,656 | 6.000000 | 3.301927 | .0277778 |
| 37 | 1369 | 50,653 | 6.082763 | 3.332222 | .0270270 |

POWERS, ROOTS, AND RECIPROCAL

(Continued)

| Number | Square | Cube | Square root | Cube root | Reciprocal |
|--------|--------|---------|-------------|-----------|------------|
| 38 | 1444 | 54,872 | 6.164414 | 3.361975 | .0263158 |
| 39 | 1521 | 59,319 | 6.244998 | 3.391211 | .0256410 |
| 40 | 1600 | 64,000 | 6.324555 | 3.419952 | .0250000 |
| 41 | 1681 | 68,921 | 6.403124 | 3.448217 | .0243902 |
| 42 | 1764 | 74,088 | 6.480741 | 3.476027 | .0238095 |
| 43 | 1849 | 79,507 | 6.557439 | 3.503398 | .0232558 |
| 44 | 1936 | 85,184 | 6.633250 | 3.530348 | .0227273 |
| 45 | 2025 | 91,125 | 6.708204 | 3.556893 | .0222222 |
| 46 | 2116 | 97,336 | 6.782330 | 3.583048 | .0217391 |
| 47 | 2209 | 103,823 | 6.855655 | 3.608826 | .0212766 |
| 48 | 2304 | 110,592 | 6.928203 | 3.634241 | .0208333 |
| 49 | 2401 | 117,649 | 7.000000 | 3.659306 | .0204082 |
| 50 | 2500 | 125,000 | 7.071068 | 3.684031 | .0200000 |
| 51 | 2601 | 132,651 | 7.141428 | 3.708430 | .0196078 |
| 52 | 2704 | 140,608 | 7.211103 | 3.732511 | .0192308 |
| 53 | 2809 | 148,877 | 7.280110 | 3.756286 | .0188679 |
| 54 | 2916 | 157,464 | 7.348469 | 3.779763 | .0185185 |
| 55 | 3025 | 166,375 | 7.416199 | 3.802953 | .0181818 |
| 56 | 3136 | 175,616 | 7.483315 | 3.825862 | .0178571 |
| 57 | 3249 | 185,193 | 7.549834 | 3.848501 | .0175439 |
| 58 | 3364 | 195,112 | 7.615773 | 3.870877 | .0172414 |
| 59 | 3481 | 205,379 | 7.681146 | 3.892997 | .0169492 |
| 60 | 3600 | 216,000 | 7.745967 | 3.914868 | .0166667 |
| 61 | 3721 | 226,981 | 7.810250 | 3.936497 | .0163934 |
| 62 | 3844 | 238,328 | 7.874008 | 3.957892 | .0161290 |
| 63 | 3969 | 250,047 | 7.937254 | 3.979057 | .0158730 |
| 64 | 4096 | 262,144 | 8.000000 | 4.000000 | .0156250 |
| 65 | 4225 | 274,625 | 8.062258 | 4.020726 | .0153846 |
| 66 | 4356 | 287,496 | 8.124038 | 4.041240 | .0151515 |
| 67 | 4489 | 300,763 | 8.185353 | 4.061548 | .0149254 |
| 68 | 4624 | 314,432 | 8.246211 | 4.081655 | .0147059 |
| 69 | 4761 | 328,509 | 8.306624 | 4.101566 | .0144928 |
| 70 | 4900 | 343,000 | 8.366600 | 4.121285 | .0142857 |
| 71 | 5041 | 357,911 | 8.426150 | 4.140818 | .0140845 |
| 72 | 5184 | 373,248 | 8.485281 | 4.160168 | .0138889 |
| 73 | 5329 | 389,017 | 8.544004 | 4.179339 | .0136986 |

POWERS, ROOTS, AND RECIPROCALS (*Continued*)

| Number | Square | Cube | Square root | Cube root | Reciprocal |
|--------|--------|-----------|-------------|-----------|------------|
| 74 | 5476 | 405,224 | 8.602325 | 4.198336 | .0135135 |
| 75 | 5625 | 421,875 | 8.660254 | 4.217163 | .0133333 |
| 76 | 5776 | 438,976 | 8.717798 | 4.235824 | .0131579 |
| 77 | 5929 | 456,533 | 8.774964 | 4.254321 | .0129870 |
| 78 | 6084 | 474,552 | 8.831761 | 4.272659 | .0128205 |
| 79 | 6241 | 493,039 | 8.888194 | 4.290840 | .0126582 |
| 80 | 6400 | 512,000 | 8.944272 | 4.308870 | .0125000 |
| 81 | 6561 | 531,441 | 9.000000 | 4.326749 | .0123457 |
| 82 | 6724 | 551,368 | 9.055385 | 4.344482 | .0121951 |
| 83 | 6889 | 571,787 | 9.110434 | 4.362071 | .0120482 |
| 84 | 7056 | 592,704 | 9.165151 | 4.379519 | .0119048 |
| 85 | 7225 | 614,125 | 9.219545 | 4.396830 | .0117647 |
| 86 | 7396 | 636,056 | 9.273619 | 4.414005 | .0116279 |
| 87 | 7569 | 658,503 | 9.327379 | 4.431048 | .0114943 |
| 88 | 7744 | 681,472 | 9.380832 | 4.447960 | .0113636 |
| 89 | 7921 | 704,969 | 9.433981 | 4.464745 | .0112360 |
| 90 | 8100 | 729,000 | 9.486833 | 4.481405 | .0111111 |
| 91 | 8281 | 753,571 | 9.539392 | 4.497941 | .0109890 |
| 92 | 8464 | 778,688 | 9.591663 | 4.514357 | .0108696 |
| 93 | 8649 | 804,357 | 9.643651 | 4.530655 | .0107527 |
| 94 | 8836 | 830,584 | 9.695360 | 4.546836 | .0106383 |
| 95 | 9025 | 857,375 | 9.746794 | 4.562903 | .0105263 |
| 96 | 9216 | 884,736 | 9.797959 | 4.578857 | .0104167 |
| 97 | 9409 | 912,673 | 9.848858 | 4.594701 | .0103093 |
| 98 | 9604 | 941,192 | 9.899495 | 4.610436 | .0102041 |
| 99 | 9801 | 970,299 | 9.949874 | 4.626065 | .0101010 |
| 100 | 10,000 | 1,000,000 | 10.000000 | 4.641589 | .0100000 |

Logarithmic Cross-section Paper

Cross-section paper the rulings of which are proportional to the logarithms of the scale is called logarithmic cross-section paper. This paper is most convenient for plotting equations with constant exponents since they are straight lines on logarithmic paper while

they are curves if plotted on ordinary graph paper, in which case they must be plotted point by point.

The chief use of logarithmic cross-section paper is for plotting equations of the form:

$$y = ax^n$$

If two pairs of values of x and y are known, the corresponding points may be plotted on logarithmic paper and joined by a straight line. The value of the coefficient a is equal to the intercept of this line on the Y -axis, and the value of the exponent n is equal to the slope of the line (that is, the tangent of the angle which the line makes with the X -axis). The reason for this is that plotting on logarithmic paper is equivalent to taking logarithms, in which case we would obtain:

$$\log y = \log a + n \log x$$

which is the equation of a straight line, $\log a$ being the intercept and n the slope.

In case the values of a and n are known, that is, the intercept and the slope, we may plot the line, and from it obtain any pair of values of x and y .

Use of Logarithm Tables

Every logarithm consists of two parts: a positive or negative whole number called the **characteristic**, and a **positive** fraction, called the **mantissa**. The mantissa is always expressed as a decimal, and is the part which is given in the tables.

To find the common logarithm of a given number:

If the number is greater than 1, the characteristic of the logarithm is one unit less than the number of figures on the left of the decimal point.

If the number is less than 1, the characteristic of the logarithm is negative, and one unit more than the number of zeros between the decimal point and the first significant figure of the given number.

Thus,

$$\begin{aligned}\log 20.6 &= 1.3139 && \text{(base 10)} \\ \log 2.06 &= 0.3139 \\ \log 0.206 &= 0.3139 - 1 = 9.3139 - 10 \\ \log 0.0206 &= 0.3139 - 2 = 8.3139 - 10\end{aligned}$$

To find the number corresponding to a given common logarithm:

If the characteristic of a given logarithm is positive, the number of figures in the integral part of the corresponding number is one more than the number of units in the characteristic.

If the characteristic is negative, the number of zeros between the decimal point and the first significant figure of the corresponding number is one less than the number of units in the characteristic.

COMMON LOGARITHMS OF NUMBERS
(Base 10)

| <i>N</i> | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|----------|--------|--------|--------|--------|--------|--------|--------|--------|--------|--------|
| 100 | 00 000 | 00 043 | 00 087 | 00 130 | 00 173 | 00 217 | 00 260 | 00 303 | 00 346 | 00 389 |
| 101 | 00 432 | 00 475 | 00 518 | 00 561 | 00 604 | 00 647 | 00 689 | 00 732 | 00 775 | 00 817 |
| 102 | 00 860 | 00 903 | 00 945 | 00 988 | 01 030 | 01 072 | 01 115 | 01 157 | 01 199 | 01 242 |
| 103 | 01 284 | 01 326 | 01 368 | 01 410 | 01 452 | 01 494 | 01 536 | 01 578 | 01 620 | 01 662 |
| 104 | 01 703 | 01 745 | 01 787 | 01 828 | 01 870 | 01 912 | 01 953 | 01 995 | 02 036 | 02 078 |
| 105 | 02 119 | 02 160 | 02 202 | 02 243 | 02 284 | 02 325 | 02 366 | 02 407 | 02 449 | 02 490 |
| 106 | 02 531 | 02 572 | 02 612 | 02 653 | 02 694 | 02 735 | 02 776 | 02 816 | 02 857 | 02 898 |
| 107 | 02 938 | 02 979 | 03 019 | 03 060 | 03 100 | 03 141 | 03 181 | 03 222 | 03 262 | 03 302 |
| 108 | 03 342 | 03 383 | 03 423 | 03 463 | 03 503 | 03 543 | 03 583 | 03 623 | 03 663 | 03 703 |
| 109 | 03 743 | 03 782 | 03 822 | 03 862 | 03 902 | 03 941 | 03 981 | 04 021 | 04 060 | 04 100 |
| 110 | 04 139 | 04 179 | 04 218 | 04 258 | 04 297 | 04 336 | 04 376 | 04 415 | 04 454 | 04 493 |
| 111 | 04 532 | 04 571 | 04 610 | 04 650 | 04 689 | 04 727 | 04 766 | 04 805 | 04 844 | 04 883 |
| 112 | 04 922 | 04 961 | 04 999 | 05 038 | 05 077 | 05 115 | 05 154 | 05 192 | 05 231 | 05 269 |
| 113 | 05 308 | 05 346 | 05 385 | 05 423 | 05 461 | 05 500 | 05 538 | 05 576 | 05 614 | 05 652 |
| 114 | 05 690 | 05 729 | 05 767 | 05 805 | 05 843 | 05 881 | 05 918 | 05 956 | 05 994 | 06 032 |
| 115 | 06 070 | 06 108 | 06 145 | 06 183 | 06 221 | 06 258 | 06 296 | 06 333 | 06 371 | 06 408 |
| 116 | 06 446 | 06 483 | 06 521 | 06 558 | 06 595 | 06 633 | 06 670 | 06 707 | 06 744 | 06 781 |
| 117 | 06 819 | 06 856 | 06 893 | 06 930 | 06 967 | 07 004 | 07 041 | 07 078 | 07 115 | 07 151 |
| 118 | 07 188 | 07 225 | 07 262 | 07 298 | 07 335 | 07 372 | 07 408 | 07 445 | 07 482 | 07 518 |
| 119 | 07 555 | 07 591 | 07 628 | 07 664 | 07 700 | 07 737 | 07 773 | 07 809 | 07 846 | 07 882 |
| 120 | 07 918 | 07 954 | 07 990 | 08 027 | 08 063 | 08 099 | 08 135 | 08 171 | 08 207 | 08 243 |
| 121 | 08 279 | 08 314 | 08 350 | 08 386 | 08 422 | 08 458 | 08 493 | 08 529 | 08 565 | 08 600 |
| 122 | 08 636 | 08 672 | 08 707 | 08 743 | 08 778 | 08 814 | 08 849 | 08 884 | 08 920 | 08 955 |
| 123 | 08 991 | 09 026 | 09 061 | 09 096 | 09 132 | 09 167 | 09 202 | 09 237 | 09 272 | 09 307 |
| 124 | 09 342 | 09 377 | 09 412 | 09 447 | 09 482 | 09 517 | 09 552 | 09 587 | 09 621 | 09 656 |
| 125 | 09 691 | 09 726 | 09 760 | 09 795 | 09 830 | 09 864 | 09 899 | 09 934 | 09 968 | 10 003 |
| 126 | 10 037 | 10 072 | 10 106 | 10 140 | 10 175 | 10 209 | 10 243 | 10 278 | 10 312 | 10 346 |
| 127 | 10 380 | 10 415 | 10 449 | 10 483 | 10 517 | 10 551 | 10 585 | 10 619 | 10 653 | 10 687 |
| 128 | 10 721 | 10 755 | 10 789 | 10 823 | 10 857 | 10 890 | 10 924 | 10 958 | 10 992 | 11 025 |
| 129 | 11 059 | 11 093 | 11 126 | 11 160 | 11 193 | 11 227 | 11 261 | 11 294 | 11 327 | 11 361 |
| 130 | 11 394 | 11 428 | 11 461 | 11 494 | 11 528 | 11 561 | 11 594 | 11 628 | 11 661 | 11 694 |
| 131 | 11 727 | 11 760 | 11 793 | 11 826 | 11 860 | 11 893 | 11 926 | 11 959 | 11 992 | 12 024 |
| 132 | 12 057 | 12 090 | 12 123 | 12 156 | 12 189 | 12 222 | 12 254 | 12 287 | 12 320 | 12 352 |
| 133 | 12 385 | 12 418 | 12 450 | 12 483 | 12 516 | 12 548 | 12 581 | 12 613 | 12 646 | 12 678 |
| 134 | 12 710 | 12 743 | 12 775 | 12 808 | 12 840 | 12 872 | 12 905 | 12 937 | 12 969 | 13 001 |
| 135 | 13 033 | 13 066 | 13 098 | 13 130 | 13 162 | 13 194 | 13 226 | 13 258 | 13 290 | 13 322 |
| 136 | 13 354 | 13 386 | 13 418 | 13 450 | 13 481 | 13 513 | 13 545 | 13 577 | 13 609 | 13 640 |
| 137 | 13 672 | 13 704 | 13 735 | 13 767 | 13 799 | 13 830 | 13 862 | 13 893 | 13 925 | 13 956 |
| 138 | 13 988 | 14 019 | 14 051 | 14 082 | 14 114 | 14 145 | 14 176 | 14 208 | 14 239 | 14 270 |
| 139 | 14 301 | 14 333 | 14 364 | 14 395 | 14 426 | 14 457 | 14 489 | 14 520 | 14 551 | 14 582 |
| 140 | 14 613 | 14 644 | 14 675 | 14 706 | 14 737 | 14 768 | 14 799 | 14 829 | 14 860 | 14 891 |
| 141 | 14 922 | 14 953 | 14 983 | 15 014 | 15 045 | 15 076 | 15 106 | 15 137 | 15 168 | 15 198 |
| 142 | 15 229 | 15 259 | 15 290 | 15 320 | 15 351 | 15 381 | 15 412 | 15 442 | 15 473 | 15 503 |
| 143 | 15 534 | 15 564 | 15 594 | 15 625 | 15 655 | 15 685 | 15 715 | 15 746 | 15 776 | 15 806 |
| 144 | 15 836 | 15 866 | 15 897 | 15 927 | 15 957 | 15 987 | 16 017 | 16 047 | 16 077 | 16 107 |

COMMON LOGARITHMS OF NUMBERS

(Continued)

| <i>N</i> | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|----------|--------|--------|--------|--------|--------|--------|--------|--------|--------|--------|
| 145 | 16 137 | 16 167 | 16 197 | 16 227 | 16 256 | 16 286 | 16 316 | 16 346 | 16 376 | 16 406 |
| 146 | 16 435 | 16 465 | 16 495 | 16 524 | 16 554 | 16 584 | 16 613 | 16 643 | 16 673 | 16 702 |
| 147 | 16 732 | 16 761 | 16 791 | 16 820 | 16 850 | 16 879 | 16 909 | 16 938 | 16 967 | 16 997 |
| 148 | 17 026 | 17 056 | 17 085 | 17 114 | 17 143 | 17 173 | 17 202 | 17 231 | 17 260 | 17 289 |
| 149 | 17 319 | 17 348 | 17 377 | 17 406 | 17 435 | 17 464 | 17 493 | 17 522 | 17 551 | 17 580 |
| 150 | 17 609 | 17 638 | 17 667 | 17 696 | 17 725 | 17 754 | 17 782 | 17 811 | 17 840 | 17 869 |
| 151 | 17 898 | 17 926 | 17 955 | 17 984 | 18 013 | 18 041 | 18 070 | 18 099 | 18 127 | 18 156 |
| 152 | 18 184 | 18 213 | 18 241 | 18 270 | 18 298 | 18 327 | 18 355 | 18 384 | 18 412 | 18 441 |
| 153 | 18 469 | 18 498 | 18 526 | 18 554 | 18 583 | 18 611 | 18 639 | 18 667 | 18 696 | 18 724 |
| 154 | 18 752 | 18 780 | 18 808 | 18 837 | 18 865 | 18 893 | 18 921 | 18 949 | 18 977 | 19 005 |
| 155 | 19 033 | 19 061 | 19 089 | 19 117 | 19 145 | 19 173 | 19 201 | 19 229 | 19 257 | 19 285 |
| 156 | 19 312 | 19 340 | 19 368 | 19 396 | 19 424 | 19 451 | 19 479 | 19 507 | 19 535 | 19 562 |
| 157 | 19 590 | 19 618 | 19 645 | 19 673 | 19 700 | 19 728 | 19 756 | 19 783 | 19 811 | 19 838 |
| 158 | 19 866 | 19 893 | 19 921 | 19 948 | 19 976 | 20 003 | 20 030 | 20 058 | 20 085 | 20 112 |
| 159 | 20 140 | 20 167 | 20 194 | 20 222 | 20 249 | 20 276 | 20 303 | 20 330 | 20 358 | 20 385 |
| 160 | 20 412 | 20 439 | 20 466 | 20 493 | 20 520 | 20 548 | 20 575 | 20 602 | 20 629 | 20 656 |
| 161 | 20 683 | 20 710 | 20 737 | 20 763 | 20 790 | 20 817 | 20 844 | 20 871 | 20 898 | 20 925 |
| 162 | 20 952 | 20 978 | 21 005 | 21 032 | 21 059 | 21 085 | 21 112 | 21 139 | 21 165 | 21 192 |
| 163 | 21 219 | 21 245 | 21 272 | 21 299 | 21 325 | 21 352 | 21 378 | 21 405 | 21 431 | 21 458 |
| 164 | 21 484 | 21 511 | 21 537 | 21 564 | 21 590 | 21 617 | 21 643 | 21 669 | 21 696 | 21 722 |
| 165 | 21 748 | 21 775 | 21 801 | 21 827 | 21 854 | 21 880 | 21 906 | 21 932 | 21 958 | 21 985 |
| 166 | 22 011 | 22 037 | 22 063 | 22 089 | 22 115 | 22 141 | 22 167 | 22 194 | 22 220 | 22 246 |
| 167 | 22 272 | 22 298 | 22 324 | 22 350 | 22 376 | 22 401 | 22 427 | 22 453 | 22 479 | 22 505 |
| 168 | 22 531 | 22 557 | 22 583 | 22 608 | 22 634 | 22 660 | 22 686 | 22 712 | 22 737 | 22 763 |
| 169 | 22 789 | 22 814 | 22 840 | 22 866 | 22 891 | 22 917 | 22 943 | 22 968 | 22 994 | 23 019 |
| 170 | 23 045 | 23 070 | 23 096 | 23 121 | 23 147 | 23 172 | 23 198 | 23 223 | 23 249 | 23 274 |
| 171 | 23 300 | 23 325 | 23 350 | 23 376 | 23 401 | 23 426 | 23 452 | 23 477 | 23 502 | 23 528 |
| 172 | 23 553 | 23 578 | 23 603 | 23 629 | 23 654 | 23 679 | 23 704 | 23 729 | 23 754 | 23 779 |
| 173 | 23 805 | 23 830 | 23 855 | 23 880 | 23 905 | 23 930 | 23 955 | 23 980 | 24 005 | 24 030 |
| 174 | 24 055 | 24 080 | 24 105 | 24 130 | 24 155 | 24 180 | 24 204 | 24 229 | 24 254 | 24 279 |
| 175 | 24 304 | 24 329 | 24 353 | 24 378 | 24 403 | 24 428 | 24 452 | 24 477 | 24 502 | 24 527 |
| 176 | 24 551 | 24 576 | 24 601 | 24 625 | 24 650 | 24 674 | 24 699 | 24 724 | 24 748 | 24 773 |
| 177 | 24 797 | 24 822 | 24 846 | 24 871 | 24 895 | 24 920 | 24 944 | 24 969 | 24 993 | 25 018 |
| 178 | 25 042 | 25 066 | 25 091 | 25 115 | 25 139 | 25 164 | 25 188 | 25 212 | 25 237 | 25 261 |
| 179 | 25 285 | 25 310 | 25 334 | 25 358 | 25 382 | 25 406 | 25 431 | 25 455 | 25 479 | 25 503 |
| 180 | 25 527 | 25 551 | 25 575 | 25 600 | 25 624 | 25 648 | 25 672 | 25 696 | 25 720 | 25 744 |
| 181 | 25 768 | 25 792 | 25 816 | 25 840 | 25 864 | 25 888 | 25 912 | 25 935 | 25 959 | 25 983 |
| 182 | 26 007 | 26 031 | 26 055 | 26 079 | 26 102 | 26 126 | 26 150 | 26 174 | 26 198 | 26 221 |
| 183 | 26 245 | 26 269 | 26 293 | 26 316 | 26 340 | 26 364 | 26 387 | 26 411 | 26 435 | 26 458 |
| 184 | 26 482 | 26 505 | 26 529 | 26 553 | 26 576 | 26 600 | 26 623 | 26 647 | 26 670 | 26 694 |
| 185 | 26 717 | 26 741 | 26 764 | 26 788 | 26 811 | 26 834 | 26 858 | 26 881 | 26 905 | 26 928 |
| 186 | 26 951 | 26 975 | 26 998 | 27 021 | 27 045 | 27 068 | 27 091 | 27 114 | 27 138 | 27 161 |
| 187 | 27 184 | 27 207 | 27 231 | 27 254 | 27 277 | 27 300 | 27 323 | 27 346 | 27 370 | 27 393 |
| 188 | 27 416 | 27 439 | 27 462 | 27 485 | 27 508 | 27 531 | 27 554 | 27 577 | 27 600 | 27 623 |
| 189 | 27 646 | 27 669 | 27 692 | 27 715 | 27 738 | 27 761 | 27 784 | 27 807 | 27 830 | 27 852 |

COMMON LOGARITHMS OF NUMBERS

(Continued)

| N | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|-----|--------|--------|--------|--------|--------|--------|--------|--------|--------|--------|
| 190 | 27 875 | 27 898 | 27 921 | 27 944 | 27 967 | 27 989 | 28 012 | 28 035 | 28 058 | 28 081 |
| 191 | 28 103 | 28 126 | 28 149 | 28 171 | 28 194 | 28 217 | 28 240 | 28 262 | 28 285 | 28 307 |
| 192 | 28 330 | 28 353 | 28 375 | 28 398 | 28 421 | 28 443 | 28 466 | 28 488 | 28 511 | 28 533 |
| 193 | 28 556 | 28 578 | 28 601 | 28 623 | 28 646 | 28 668 | 28 691 | 28 713 | 28 735 | 28 758 |
| 194 | 28 780 | 28 803 | 28 825 | 28 847 | 28 870 | 28 892 | 28 914 | 28 937 | 28 959 | 28 981 |
| 195 | 29 003 | 29 026 | 29 048 | 29 070 | 29 092 | 29 115 | 29 137 | 29 159 | 29 181 | 29 203 |
| 196 | 29 226 | 29 248 | 29 270 | 29 292 | 29 314 | 29 336 | 29 358 | 29 380 | 29 403 | 29 425 |
| 197 | 29 447 | 29 469 | 29 491 | 29 513 | 29 535 | 29 557 | 29 579 | 29 601 | 29 623 | 29 645 |
| 198 | 29 667 | 29 688 | 29 710 | 29 732 | 29 754 | 29 776 | 29 798 | 29 820 | 29 842 | 29 863 |
| 199 | 29 885 | 29 907 | 29 929 | 29 951 | 29 973 | 29 994 | 30 016 | 30 038 | 30 060 | 30 081 |
| 200 | 30 103 | 30 125 | 30 146 | 30 168 | 30 190 | 30 211 | 30 233 | 30 255 | 30 276 | 30 298 |
| 201 | 30 320 | 30 341 | 30 363 | 30 384 | 30 406 | 30 428 | 30 449 | 30 471 | 30 492 | 30 514 |
| 202 | 30 535 | 30 557 | 30 578 | 30 600 | 30 621 | 30 643 | 30 664 | 30 685 | 30 707 | 30 728 |
| 203 | 30 750 | 30 771 | 30 792 | 30 814 | 30 835 | 30 856 | 30 878 | 30 899 | 30 920 | 30 942 |
| 204 | 30 963 | 30 984 | 31 006 | 31 027 | 31 048 | 31 069 | 31 091 | 31 112 | 31 133 | 31 154 |
| 205 | 31 175 | 31 197 | 31 218 | 31 239 | 31 260 | 31 281 | 31 302 | 31 323 | 31 345 | 31 366 |
| 206 | 31 387 | 31 408 | 31 429 | 31 450 | 31 471 | 31 492 | 31 513 | 31 534 | 31 555 | 31 576 |
| 207 | 31 597 | 31 618 | 31 639 | 31 660 | 31 681 | 31 702 | 31 723 | 31 744 | 31 765 | 31 785 |
| 208 | 31 806 | 31 827 | 31 848 | 31 869 | 31 890 | 31 911 | 31 931 | 31 952 | 31 973 | 31 994 |
| 209 | 32 015 | 32 035 | 32 056 | 32 077 | 32 098 | 32 118 | 32 139 | 32 160 | 32 181 | 32 201 |
| 210 | 32 222 | 32 243 | 32 263 | 32 284 | 32 305 | 32 325 | 32 346 | 32 366 | 32 387 | 32 408 |
| 211 | 32 428 | 32 449 | 32 469 | 32 490 | 32 510 | 32 531 | 32 552 | 32 572 | 32 593 | 32 613 |
| 212 | 32 634 | 32 654 | 32 675 | 32 695 | 32 715 | 32 736 | 32 756 | 32 777 | 32 797 | 32 818 |
| 213 | 32 838 | 32 858 | 32 879 | 32 899 | 32 919 | 32 940 | 32 960 | 32 980 | 33 001 | 33 021 |
| 214 | 33 041 | 33 062 | 33 082 | 33 102 | 33 122 | 33 143 | 33 163 | 33 183 | 33 203 | 33 224 |
| 215 | 33 244 | 33 264 | 33 284 | 33 304 | 33 325 | 33 345 | 33 365 | 33 385 | 33 405 | 33 425 |
| 216 | 33 445 | 33 465 | 33 486 | 33 506 | 33 526 | 33 546 | 33 566 | 33 586 | 33 606 | 33 626 |
| 217 | 33 646 | 33 666 | 33 686 | 33 706 | 33 726 | 33 746 | 33 766 | 33 786 | 33 806 | 33 826 |
| 218 | 33 846 | 33 866 | 33 885 | 33 905 | 33 925 | 33 945 | 33 965 | 33 985 | 34 005 | 34 025 |
| 219 | 34 044 | 34 064 | 34 084 | 34 104 | 34 124 | 34 143 | 34 163 | 34 183 | 34 203 | 34 223 |
| 220 | 34 242 | 34 262 | 34 282 | 34 301 | 34 321 | 34 341 | 34 361 | 34 380 | 34 400 | 34 420 |
| 221 | 34 439 | 34 459 | 34 479 | 34 498 | 34 518 | 34 537 | 34 557 | 34 577 | 34 596 | 34 616 |
| 222 | 34 635 | 34 655 | 34 674 | 34 694 | 34 713 | 34 733 | 34 753 | 34 772 | 34 792 | 34 811 |
| 223 | 34 830 | 34 850 | 34 869 | 34 889 | 34 908 | 34 928 | 34 947 | 34 967 | 34 986 | 35 005 |
| 224 | 35 025 | 35 044 | 35 064 | 35 083 | 35 102 | 35 122 | 35 141 | 35 160 | 35 180 | 35 199 |
| 225 | 35 218 | 35 238 | 35 257 | 35 276 | 35 295 | 35 315 | 35 334 | 35 353 | 35 372 | 35 392 |
| 226 | 35 411 | 35 430 | 35 449 | 35 468 | 35 488 | 35 507 | 35 526 | 35 545 | 35 564 | 35 583 |
| 227 | 35 603 | 35 622 | 35 641 | 35 660 | 35 679 | 35 698 | 35 717 | 35 736 | 35 755 | 35 774 |
| 228 | 35 793 | 35 813 | 35 832 | 35 851 | 35 870 | 35 889 | 35 908 | 35 927 | 35 946 | 35 965 |
| 229 | 35 984 | 36 003 | 36 021 | 36 040 | 36 059 | 36 078 | 36 097 | 36 116 | 36 135 | 36 154 |
| 230 | 36 173 | 36 192 | 36 211 | 36 229 | 36 248 | 36 267 | 36 286 | 36 305 | 36 324 | 36 342 |
| 231 | 36 361 | 36 380 | 36 399 | 36 418 | 36 436 | 36 455 | 36 474 | 36 493 | 36 511 | 36 530 |
| 232 | 36 549 | 36 568 | 36 586 | 36 605 | 36 624 | 36 642 | 36 661 | 36 680 | 36 698 | 36 717 |
| 233 | 36 736 | 36 754 | 36 773 | 36 791 | 36 810 | 36 829 | 36 847 | 36 866 | 36 884 | 36 903 |
| 234 | 36 922 | 36 940 | 36 959 | 36 977 | 36 996 | 37 014 | 37 033 | 37 051 | 37 070 | 37 088 |

COMMON LOGARITHMS OF NUMBERS

(Continued)

| N | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|-----|--------|--------|--------|--------|--------|--------|--------|--------|--------|--------|
| 235 | 37 107 | 37 125 | 37 144 | 37 162 | 37 181 | 37 199 | 37 218 | 37 236 | 37 254 | 37 273 |
| 236 | 37 291 | 37 310 | 37 328 | 37 346 | 37 365 | 37 383 | 37 401 | 37 420 | 37 438 | 37 457 |
| 237 | 37 475 | 37 493 | 37 511 | 37 530 | 37 548 | 37 566 | 37 585 | 37 603 | 37 621 | 37 639 |
| 238 | 37 658 | 37 676 | 37 694 | 37 712 | 37 731 | 37 749 | 37 767 | 37 785 | 37 803 | 37 822 |
| 239 | 37 840 | 37 858 | 37 876 | 37 894 | 37 912 | 37 931 | 37 949 | 37 967 | 37 985 | 38 003 |
| 240 | 38 021 | 38 039 | 38 057 | 38 075 | 38 093 | 38 112 | 38 130 | 38 148 | 38 166 | 38 184 |
| 241 | 38 202 | 38 220 | 38 238 | 38 256 | 38 274 | 38 292 | 38 310 | 38 328 | 38 346 | 38 364 |
| 242 | 38 382 | 38 399 | 38 417 | 38 435 | 38 453 | 38 471 | 38 489 | 38 507 | 38 525 | 38 543 |
| 243 | 38 561 | 38 578 | 38 596 | 38 614 | 38 632 | 38 650 | 38 668 | 38 686 | 38 703 | 38 721 |
| 244 | 38 739 | 38 757 | 38 775 | 38 792 | 38 810 | 38 828 | 38 846 | 38 863 | 38 881 | 38 899 |
| 245 | 38 917 | 38 934 | 38 952 | 38 970 | 38 987 | 39 005 | 39 023 | 39 041 | 39 058 | 39 076 |
| 246 | 39 094 | 39 111 | 39 129 | 39 146 | 39 164 | 39 182 | 39 199 | 39 217 | 39 235 | 39 252 |
| 247 | 39 270 | 39 287 | 39 305 | 39 322 | 39 340 | 39 358 | 39 375 | 39 393 | 39 410 | 39 428 |
| 248 | 39 445 | 39 463 | 39 480 | 39 498 | 39 515 | 39 533 | 39 550 | 39 568 | 39 585 | 39 602 |
| 249 | 39 620 | 39 637 | 39 655 | 39 672 | 39 690 | 39 707 | 39 724 | 39 742 | 39 759 | 39 777 |
| 250 | 39 794 | 39 811 | 39 829 | 39 846 | 39 863 | 39 881 | 39 898 | 39 915 | 39 933 | 39 950 |
| 251 | 39 967 | 39 985 | 40 002 | 40 019 | 40 037 | 40 054 | 40 071 | 40 088 | 40 106 | 40 123 |
| 252 | 40 140 | 40 157 | 40 175 | 40 192 | 40 209 | 40 226 | 40 243 | 40 261 | 40 278 | 40 295 |
| 253 | 40 312 | 40 329 | 40 346 | 40 364 | 40 381 | 40 398 | 40 415 | 40 432 | 40 449 | 40 466 |
| 254 | 40 483 | 40 500 | 40 518 | 40 535 | 40 552 | 40 569 | 40 586 | 40 603 | 40 620 | 40 637 |
| 255 | 40 654 | 40 671 | 40 688 | 40 705 | 40 722 | 40 739 | 40 756 | 40 773 | 40 790 | 40 807 |
| 256 | 40 824 | 40 841 | 40 858 | 40 875 | 40 892 | 40 909 | 40 926 | 40 943 | 40 960 | 40 976 |
| 257 | 40 993 | 41 010 | 41 027 | 41 044 | 41 061 | 41 078 | 41 095 | 41 111 | 41 128 | 41 145 |
| 258 | 41 162 | 41 179 | 41 196 | 41 212 | 41 229 | 41 246 | 41 263 | 41 280 | 41 296 | 41 313 |
| 259 | 41 330 | 41 347 | 41 363 | 41 380 | 41 397 | 41 414 | 41 430 | 41 447 | 41 464 | 41 481 |
| 260 | 41 497 | 41 514 | 41 531 | 41 547 | 41 564 | 41 581 | 41 597 | 41 614 | 41 631 | 41 647 |
| 261 | 41 664 | 41 681 | 41 697 | 41 714 | 41 731 | 41 747 | 41 764 | 41 780 | 41 797 | 41 814 |
| 262 | 41 830 | 41 847 | 41 863 | 41 880 | 41 896 | 41 913 | 41 929 | 41 946 | 41 963 | 41 979 |
| 263 | 41 996 | 42 012 | 42 029 | 42 045 | 42 062 | 42 078 | 42 095 | 42 111 | 42 127 | 42 144 |
| 264 | 42 160 | 42 177 | 42 193 | 42 210 | 42 226 | 42 243 | 42 259 | 42 275 | 42 292 | 42 308 |
| 265 | 42 325 | 42 341 | 42 357 | 42 374 | 42 390 | 42 406 | 42 423 | 42 439 | 42 455 | 42 472 |
| 266 | 42 488 | 42 504 | 42 521 | 42 537 | 42 553 | 42 570 | 42 586 | 42 602 | 42 619 | 42 635 |
| 267 | 42 651 | 42 667 | 42 684 | 42 700 | 42 716 | 42 732 | 42 749 | 42 765 | 42 781 | 42 797 |
| 268 | 42 813 | 42 830 | 42 846 | 42 862 | 42 878 | 42 894 | 42 911 | 42 927 | 42 943 | 42 959 |
| 269 | 42 975 | 42 991 | 43 008 | 43 024 | 43 040 | 43 056 | 43 072 | 43 088 | 43 104 | 43 120 |
| 270 | 43 136 | 43 152 | 43 169 | 43 185 | 43 201 | 43 217 | 43 233 | 43 249 | 43 265 | 43 281 |
| 271 | 43 297 | 43 313 | 43 329 | 43 345 | 43 361 | 43 377 | 43 393 | 43 409 | 43 425 | 43 441 |
| 272 | 43 457 | 43 473 | 43 489 | 43 505 | 43 521 | 43 537 | 43 553 | 43 569 | 43 584 | 43 600 |
| 273 | 43 616 | 43 632 | 43 648 | 43 664 | 43 680 | 43 696 | 43 712 | 43 727 | 43 743 | 43 759 |
| 274 | 43 775 | 43 791 | 43 807 | 43 823 | 43 838 | 43 854 | 43 870 | 43 886 | 43 902 | 43 917 |
| 275 | 43 933 | 43 949 | 43 965 | 43 981 | 43 996 | 44 012 | 44 028 | 44 044 | 44 059 | 44 075 |
| 276 | 44 091 | 44 107 | 44 122 | 44 138 | 44 154 | 44 170 | 44 185 | 44 201 | 44 217 | 44 232 |
| 277 | 44 248 | 44 264 | 44 279 | 44 295 | 44 311 | 44 326 | 44 342 | 44 358 | 44 373 | 44 389 |
| 278 | 44 404 | 44 420 | 44 436 | 44 451 | 44 467 | 44 483 | 44 498 | 44 514 | 44 529 | 44 545 |
| 279 | 44 560 | 44 576 | 44 592 | 44 607 | 44 623 | 44 638 | 44 654 | 44 669 | 44 685 | 44 700 |

COMMON LOGARITHMS OF NUMBERS

(Continued)

| N | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|-----|--------|--------|--------|--------|--------|--------|--------|--------|--------|--------|
| 280 | 44 716 | 44 731 | 44 747 | 44 762 | 44 778 | 44 793 | 44 809 | 44 824 | 44 840 | 44 855 |
| 281 | 44 871 | 44 886 | 44 902 | 44 917 | 44 932 | 44 948 | 44 963 | 44 979 | 44 994 | 45 010 |
| 282 | 45 025 | 45 040 | 45 056 | 45 071 | 45 086 | 45 102 | 45 117 | 45 133 | 45 148 | 45 163 |
| 283 | 45 179 | 45 194 | 45 209 | 45 225 | 45 240 | 45 255 | 45 271 | 45 286 | 45 301 | 45 317 |
| 284 | 45 332 | 45 347 | 45 362 | 45 378 | 45 393 | 45 408 | 45 423 | 45 439 | 45 454 | 45 469 |
| 285 | 45 484 | 45 500 | 45 515 | 45 530 | 45 545 | 45 561 | 45 576 | 45 591 | 45 606 | 45 621 |
| 286 | 45 637 | 45 652 | 45 667 | 45 682 | 45 697 | 45 712 | 45 728 | 45 743 | 45 758 | 45 773 |
| 287 | 45 788 | 45 803 | 45 818 | 45 834 | 45 849 | 45 864 | 45 879 | 45 894 | 45 909 | 45 924 |
| 288 | 45 939 | 45 954 | 45 969 | 45 984 | 46 000 | 46 015 | 46 030 | 46 045 | 46 060 | 46 075 |
| 289 | 46 090 | 46 105 | 46 120 | 46 135 | 46 150 | 46 165 | 46 180 | 46 195 | 46 210 | 46 225 |
| 290 | 46 240 | 46 255 | 46 270 | 46 285 | 46 300 | 46 315 | 46 330 | 46 345 | 46 359 | 46 374 |
| 291 | 46 389 | 46 404 | 46 419 | 46 434 | 46 449 | 46 464 | 46 479 | 46 494 | 46 509 | 46 523 |
| 292 | 46 538 | 46 553 | 46 568 | 46 583 | 46 598 | 46 613 | 46 627 | 46 642 | 46 657 | 46 672 |
| 293 | 46 687 | 46 702 | 46 716 | 46 731 | 46 746 | 46 761 | 46 776 | 46 790 | 46 805 | 46 820 |
| 294 | 46 835 | 46 850 | 46 864 | 46 879 | 46 894 | 46 909 | 46 923 | 46 938 | 46 953 | 46 967 |
| 295 | 46 982 | 46 997 | 47 012 | 47 026 | 47 041 | 47 056 | 47 070 | 47 085 | 47 100 | 47 114 |
| 296 | 47 129 | 47 144 | 47 159 | 47 173 | 47 188 | 47 202 | 47 217 | 47 232 | 47 246 | 47 261 |
| 297 | 47 276 | 47 290 | 47 305 | 47 319 | 47 334 | 47 349 | 47 363 | 47 378 | 47 392 | 47 407 |
| 298 | 47 422 | 47 436 | 47 451 | 47 465 | 47 480 | 47 494 | 47 509 | 47 524 | 47 538 | 47 553 |
| 299 | 47 567 | 47 582 | 47 596 | 47 611 | 47 625 | 47 640 | 47 654 | 47 669 | 47 683 | 47 698 |
| 300 | 47 712 | 47 727 | 47 741 | 47 756 | 47 770 | 47 784 | 47 799 | 47 813 | 47 828 | 47 842 |
| 301 | 47 857 | 47 871 | 47 885 | 47 900 | 47 914 | 47 929 | 47 943 | 47 958 | 47 972 | 47 986 |
| 302 | 48 001 | 48 015 | 48 029 | 48 044 | 48 058 | 48 073 | 48 087 | 48 101 | 48 116 | 48 130 |
| 303 | 48 144 | 48 159 | 48 173 | 48 187 | 48 202 | 48 216 | 48 230 | 48 244 | 48 259 | 48 273 |
| 304 | 48 287 | 48 302 | 48 316 | 48 330 | 48 344 | 48 359 | 48 373 | 48 387 | 48 401 | 48 416 |
| 305 | 48 430 | 48 444 | 48 458 | 48 473 | 48 487 | 48 501 | 48 515 | 48 530 | 48 544 | 48 558 |
| 306 | 48 572 | 48 586 | 48 601 | 48 615 | 48 629 | 48 643 | 48 657 | 48 671 | 48 686 | 48 700 |
| 307 | 48 714 | 48 728 | 48 742 | 48 756 | 48 770 | 48 785 | 48 799 | 48 813 | 48 827 | 48 841 |
| 308 | 48 855 | 48 869 | 48 883 | 48 897 | 48 911 | 48 926 | 48 940 | 48 954 | 48 968 | 48 982 |
| 309 | 48 996 | 49 010 | 49 024 | 49 038 | 49 052 | 49 066 | 49 080 | 49 094 | 49 108 | 49 122 |
| 310 | 49 136 | 49 150 | 49 164 | 49 178 | 49 192 | 49 206 | 49 220 | 49 234 | 49 248 | 49 262 |
| 311 | 49 276 | 49 290 | 49 304 | 49 318 | 49 332 | 49 346 | 49 360 | 49 374 | 49 388 | 49 402 |
| 312 | 49 415 | 49 429 | 49 443 | 49 457 | 49 471 | 49 485 | 49 499 | 49 513 | 49 527 | 49 541 |
| 313 | 49 554 | 49 568 | 49 582 | 49 596 | 49 610 | 49 624 | 49 638 | 49 651 | 49 665 | 49 679 |
| 314 | 49 693 | 49 707 | 49 721 | 49 734 | 49 748 | 49 762 | 49 776 | 49 790 | 49 803 | 49 817 |
| 315 | 49 831 | 49 845 | 49 859 | 49 872 | 49 886 | 49 900 | 49 914 | 49 927 | 49 941 | 49 955 |
| 316 | 49 969 | 49 982 | 49 996 | 50 010 | 50 024 | 50 037 | 50 051 | 50 065 | 50 079 | 50 092 |
| 317 | 50 106 | 50 120 | 50 133 | 50 147 | 50 161 | 50 174 | 50 188 | 50 202 | 50 215 | 50 229 |
| 318 | 50 243 | 50 256 | 50 270 | 50 284 | 50 297 | 50 311 | 50 325 | 50 338 | 50 352 | 50 365 |
| 319 | 50 379 | 50 393 | 50 406 | 50 420 | 50 433 | 50 447 | 50 461 | 50 474 | 50 488 | 50 501 |
| 320 | 50 515 | 50 529 | 50 542 | 50 556 | 50 569 | 50 583 | 50 596 | 50 610 | 50 623 | 50 637 |
| 321 | 50 651 | 50 664 | 50 678 | 50 691 | 50 705 | 50 718 | 50 732 | 50 745 | 50 759 | 50 772 |
| 322 | 50 786 | 50 799 | 50 813 | 50 826 | 50 840 | 50 853 | 50 866 | 50 880 | 50 893 | 50 907 |
| 323 | 50 920 | 50 934 | 50 947 | 50 961 | 50 974 | 50 987 | 51 001 | 51 014 | 51 028 | 51 041 |
| 324 | 51 055 | 51 068 | 51 081 | 51 095 | 51 108 | 51 121 | 51 135 | 51 148 | 51 162 | 51 175 |

COMMON LOGARITHMS OF NUMBERS

(Continued)

| N | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|-----|--------|--------|--------|--------|--------|--------|--------|--------|--------|--------|
| 325 | 51 188 | 51 202 | 51 215 | 51 228 | 51 242 | 51 255 | 51 268 | 51 282 | 51 295 | 51 308 |
| 326 | 51 322 | 51 335 | 51 348 | 51 362 | 51 375 | 51 388 | 51 402 | 51 415 | 51 428 | 51 441 |
| 327 | 51 455 | 51 468 | 51 481 | 51 495 | 51 508 | 51 521 | 51 534 | 51 548 | 51 561 | 51 574 |
| 328 | 51 587 | 51 601 | 51 614 | 51 627 | 51 640 | 51 654 | 51 667 | 51 680 | 51 693 | 51 706 |
| 329 | 51 720 | 51 733 | 51 746 | 51 759 | 51 772 | 51 786 | 51 799 | 51 812 | 51 825 | 51 838 |
| 330 | 51 851 | 51 865 | 51 878 | 51 891 | 51 904 | 51 917 | 51 930 | 51 943 | 51 957 | 51 970 |
| 331 | 51 983 | 51 996 | 52 009 | 52 022 | 52 035 | 52 048 | 52 061 | 52 075 | 52 088 | 52 101 |
| 332 | 52 114 | 52 127 | 52 140 | 52 153 | 52 166 | 52 179 | 52 192 | 52 205 | 52 218 | 52 231 |
| 333 | 52 244 | 52 257 | 52 270 | 52 284 | 52 297 | 52 310 | 52 323 | 52 336 | 52 349 | 52 362 |
| 334 | 52 375 | 52 388 | 52 401 | 52 414 | 52 427 | 52 440 | 52 453 | 52 466 | 52 479 | 52 492 |
| 335 | 52 504 | 52 517 | 52 530 | 52 543 | 52 556 | 52 569 | 52 582 | 52 595 | 52 608 | 52 621 |
| 336 | 52 634 | 52 647 | 52 660 | 52 673 | 52 686 | 52 699 | 52 711 | 52 724 | 52 737 | 52 750 |
| 337 | 52 763 | 52 776 | 52 789 | 52 802 | 52 815 | 52 827 | 52 840 | 52 853 | 52 866 | 52 879 |
| 338 | 52 892 | 52 905 | 52 917 | 52 930 | 52 943 | 52 956 | 52 969 | 52 982 | 52 994 | 53 007 |
| 339 | 53 020 | 53 033 | 53 046 | 53 058 | 53 071 | 53 084 | 53 097 | 53 110 | 53 122 | 53 135 |
| 340 | 53 148 | 53 161 | 53 173 | 53 186 | 53 199 | 53 212 | 53 224 | 53 237 | 53 250 | 53 263 |
| 341 | 53 275 | 53 288 | 53 301 | 53 314 | 53 326 | 53 339 | 53 352 | 53 364 | 53 377 | 53 390 |
| 342 | 53 403 | 53 415 | 53 428 | 53 441 | 53 453 | 53 466 | 53 479 | 53 491 | 53 504 | 53 517 |
| 343 | 53 529 | 53 542 | 53 555 | 53 567 | 53 580 | 53 593 | 53 605 | 53 618 | 53 631 | 53 643 |
| 344 | 53 656 | 53 668 | 53 681 | 53 694 | 53 706 | 53 719 | 53 732 | 53 744 | 53 757 | 53 769 |
| 345 | 53 782 | 53 794 | 53 807 | 53 820 | 53 832 | 53 845 | 53 857 | 53 870 | 53 882 | 53 895 |
| 346 | 53 908 | 53 920 | 53 933 | 53 945 | 53 958 | 53 970 | 53 983 | 53 995 | 54 008 | 54 020 |
| 347 | 54 033 | 54 045 | 54 058 | 54 070 | 54 083 | 54 095 | 54 108 | 54 120 | 54 133 | 54 145 |
| 348 | 54 158 | 54 170 | 54 183 | 54 195 | 54 208 | 54 220 | 54 233 | 54 245 | 54 258 | 54 270 |
| 349 | 54 283 | 54 295 | 54 307 | 54 320 | 54 332 | 54 345 | 54 357 | 54 370 | 54 382 | 54 394 |
| 350 | 54 407 | 54 419 | 54 432 | 54 444 | 54 456 | 54 469 | 54 481 | 54 494 | 54 506 | 54 518 |
| 351 | 54 531 | 54 543 | 54 555 | 54 568 | 54 580 | 54 593 | 54 605 | 54 617 | 54 630 | 54 642 |
| 352 | 54 654 | 54 667 | 54 679 | 54 691 | 54 704 | 54 716 | 54 728 | 54 741 | 54 753 | 54 765 |
| 353 | 54 777 | 54 790 | 54 802 | 54 814 | 54 827 | 54 839 | 54 851 | 54 864 | 54 876 | 54 888 |
| 354 | 54 900 | 54 913 | 54 925 | 54 937 | 54 949 | 54 962 | 54 974 | 54 986 | 54 998 | 55 011 |
| 355 | 55 023 | 55 035 | 55 047 | 55 060 | 55 072 | 55 084 | 55 096 | 55 108 | 55 121 | 55 133 |
| 356 | 55 145 | 55 157 | 55 169 | 55 182 | 55 194 | 55 206 | 55 218 | 55 230 | 55 242 | 55 255 |
| 357 | 55 267 | 55 279 | 55 291 | 55 303 | 55 315 | 55 328 | 55 340 | 55 352 | 55 364 | 55 376 |
| 358 | 55 388 | 55 400 | 55 413 | 55 425 | 55 437 | 55 449 | 55 461 | 55 473 | 55 485 | 55 497 |
| 359 | 55 509 | 55 522 | 55 534 | 55 546 | 55 558 | 55 570 | 55 582 | 55 594 | 55 606 | 55 618 |
| 360 | 55 630 | 55 642 | 55 654 | 55 666 | 55 678 | 55 691 | 55 703 | 55 715 | 55 727 | 55 739 |
| 361 | 55 751 | 55 763 | 55 775 | 55 787 | 55 799 | 55 811 | 55 823 | 55 835 | 55 847 | 55 859 |
| 362 | 55 871 | 55 883 | 55 895 | 55 907 | 55 919 | 55 931 | 55 943 | 55 955 | 55 967 | 55 979 |
| 363 | 55 991 | 56 003 | 56 015 | 56 027 | 56 038 | 56 050 | 56 062 | 56 074 | 56 086 | 56 098 |
| 364 | 56 110 | 56 122 | 56 134 | 56 146 | 56 158 | 56 170 | 56 182 | 56 194 | 56 205 | 56 217 |
| 365 | 56 229 | 56 241 | 56 253 | 56 265 | 56 277 | 56 289 | 56 301 | 56 312 | 56 324 | 56 336 |
| 366 | 56 348 | 56 360 | 56 372 | 56 384 | 56 396 | 56 407 | 56 419 | 56 431 | 56 443 | 56 455 |
| 367 | 56 467 | 56 478 | 56 490 | 56 502 | 56 514 | 56 526 | 56 538 | 56 549 | 56 561 | 56 573 |
| 368 | 56 585 | 56 597 | 56 608 | 56 620 | 56 632 | 56 644 | 56 656 | 56 667 | 56 679 | 56 691 |
| 369 | 56 703 | 56 714 | 56 726 | 56 738 | 56 750 | 56 761 | 56 773 | 56 785 | 56 797 | 56 808 |

COMMON LOGARITHMS OF NUMBERS

(Continued)

| N | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|-----|--------|--------|--------|--------|--------|--------|--------|--------|--------|--------|
| 370 | 56 820 | 56 832 | 56 844 | 56 855 | 56 867 | 56 879 | 56 891 | 56 902 | 56 914 | 56 926 |
| 371 | 56 937 | 56 949 | 56 961 | 56 972 | 56 984 | 56 996 | 57 008 | 57 019 | 57 031 | 57 043 |
| 372 | 57 054 | 57 066 | 57 078 | 57 089 | 57 101 | 57 113 | 57 124 | 57 136 | 57 148 | 57 159 |
| 373 | 57 171 | 57 183 | 57 194 | 57 206 | 57 217 | 57 229 | 57 241 | 57 252 | 57 264 | 57 276 |
| 374 | 57 287 | 57 299 | 57 310 | 57 322 | 57 334 | 57 345 | 57 357 | 57 368 | 57 380 | 57 392 |
| 375 | 57 403 | 57 415 | 57 426 | 57 438 | 57 449 | 57 461 | 57 473 | 57 484 | 57 496 | 57 507 |
| 376 | 57 519 | 57 530 | 57 542 | 57 553 | 57 565 | 57 576 | 57 588 | 57 600 | 57 611 | 57 623 |
| 377 | 57 634 | 57 646 | 57 657 | 57 669 | 57 680 | 57 692 | 57 703 | 57 715 | 57 726 | 57 738 |
| 378 | 57 749 | 57 761 | 57 772 | 57 784 | 57 795 | 57 807 | 57 818 | 57 830 | 57 841 | 57 852 |
| 379 | 57 864 | 57 875 | 57 887 | 57 898 | 57 910 | 57 921 | 57 933 | 57 944 | 57 955 | 57 967 |
| 380 | 57 978 | 57 990 | 58 001 | 58 013 | 58 024 | 58 035 | 58 047 | 58 058 | 58 070 | 58 081 |
| 381 | 58 092 | 58 104 | 58 115 | 58 127 | 58 138 | 58 149 | 58 161 | 58 172 | 58 184 | 58 195 |
| 382 | 58 206 | 58 218 | 58 229 | 58 240 | 58 252 | 58 263 | 58 274 | 58 286 | 58 297 | 58 309 |
| 383 | 58 320 | 58 331 | 58 343 | 58 354 | 58 365 | 58 377 | 58 388 | 58 399 | 58 410 | 58 422 |
| 384 | 58 433 | 58 444 | 58 456 | 58 467 | 58 478 | 58 490 | 58 501 | 58 512 | 58 524 | 58 535 |
| 385 | 58 546 | 58 557 | 58 569 | 58 580 | 58 591 | 58 602 | 58 614 | 58 625 | 58 636 | 58 647 |
| 386 | 58 659 | 58 670 | 58 681 | 58 692 | 58 704 | 58 715 | 58 726 | 58 737 | 58 749 | 58 760 |
| 387 | 58 771 | 58 782 | 58 794 | 58 805 | 58 816 | 58 827 | 58 838 | 58 850 | 58 861 | 58 872 |
| 388 | 58 883 | 58 894 | 58 906 | 58 917 | 58 928 | 58 939 | 58 950 | 58 961 | 58 973 | 58 984 |
| 389 | 58 995 | 59 006 | 59 017 | 59 028 | 59 040 | 59 051 | 59 062 | 59 073 | 59 084 | 59 095 |
| 390 | 59 106 | 59 118 | 59 129 | 59 140 | 59 151 | 59 162 | 59 173 | 59 184 | 59 195 | 59 207 |
| 391 | 59 218 | 59 229 | 59 240 | 59 251 | 59 262 | 59 273 | 59 284 | 59 295 | 59 306 | 59 318 |
| 392 | 59 329 | 59 340 | 59 351 | 59 362 | 59 373 | 59 384 | 59 395 | 59 406 | 59 417 | 59 428 |
| 393 | 59 439 | 59 450 | 59 461 | 59 472 | 59 483 | 59 494 | 59 506 | 59 517 | 59 528 | 59 539 |
| 394 | 59 550 | 59 561 | 59 572 | 59 583 | 59 594 | 59 605 | 59 616 | 59 627 | 59 638 | 59 649 |
| 395 | 59 660 | 59 671 | 59 682 | 59 693 | 59 704 | 59 715 | 59 726 | 59 737 | 59 748 | 59 759 |
| 396 | 59 770 | 59 780 | 59 791 | 59 802 | 59 813 | 59 824 | 59 835 | 59 846 | 59 857 | 59 868 |
| 397 | 59 879 | 59 890 | 59 901 | 59 912 | 59 923 | 59 934 | 59 945 | 59 956 | 59 966 | 59 977 |
| 398 | 59 988 | 59 999 | 60 010 | 60 021 | 60 032 | 60 043 | 60 054 | 60 065 | 60 076 | 60 086 |
| 399 | 60 097 | 60 108 | 60 119 | 60 130 | 60 141 | 60 152 | 60 163 | 60 173 | 60 184 | 60 195 |
| 400 | 60 206 | 60 217 | 60 228 | 60 239 | 60 249 | 60 260 | 60 271 | 60 282 | 60 293 | 60 304 |
| 401 | 60 314 | 60 325 | 60 336 | 60 347 | 60 358 | 60 369 | 60 379 | 60 390 | 60 401 | 60 412 |
| 402 | 60 423 | 60 433 | 60 444 | 60 455 | 60 466 | 60 477 | 60 487 | 60 498 | 60 509 | 60 520 |
| 403 | 60 531 | 60 541 | 60 552 | 60 563 | 60 574 | 60 584 | 60 595 | 60 606 | 60 617 | 60 627 |
| 404 | 60 638 | 60 649 | 60 660 | 60 670 | 60 681 | 60 692 | 60 703 | 60 713 | 60 724 | 60 735 |
| 405 | 60 746 | 60 756 | 60 767 | 60 778 | 60 788 | 60 799 | 60 810 | 60 821 | 60 831 | 60 842 |
| 406 | 60 853 | 60 863 | 60 874 | 60 885 | 60 895 | 60 906 | 60 917 | 60 927 | 60 938 | 60 949 |
| 407 | 60 959 | 60 970 | 60 981 | 60 991 | 61 002 | 61 013 | 61 023 | 61 034 | 61 045 | 61 055 |
| 408 | 61 066 | 61 077 | 61 087 | 61 098 | 61 109 | 61 119 | 61 130 | 61 140 | 61 151 | 61 162 |
| 409 | 61 172 | 61 183 | 61 194 | 61 204 | 61 215 | 61 225 | 61 236 | 61 247 | 61 257 | 61 268 |
| 410 | 61 278 | 61 289 | 61 300 | 61 310 | 61 321 | 61 331 | 61 342 | 61 352 | 61 363 | 61 374 |
| 411 | 61 384 | 61 395 | 61 405 | 61 416 | 61 426 | 61 437 | 61 448 | 61 458 | 61 469 | 61 479 |
| 412 | 61 490 | 61 500 | 61 511 | 61 521 | 61 532 | 61 542 | 61 553 | 61 563 | 61 574 | 61 584 |
| 413 | 61 595 | 61 606 | 61 616 | 61 627 | 61 637 | 61 648 | 61 658 | 61 669 | 61 679 | 61 690 |
| 414 | 61 700 | 61 711 | 61 721 | 61 731 | 61 742 | 61 752 | 61 763 | 61 773 | 61 784 | 61 794 |

COMMON LOGARITHMS OF NUMBERS

(Continued)

| N | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|-----|--------|--------|--------|--------|--------|--------|--------|--------|--------|--------|
| 415 | 61 805 | 61 815 | 61 826 | 61 836 | 61 847 | 61 857 | 61 868 | 61 878 | 61 888 | 61 899 |
| 416 | 61 909 | 61 920 | 61 930 | 61 941 | 61 951 | 61 962 | 61 972 | 61 982 | 61 993 | 62 003 |
| 417 | 62 014 | 62 024 | 62 034 | 62 045 | 62 055 | 62 066 | 62 076 | 62 086 | 62 097 | 62 107 |
| 418 | 62 118 | 62 128 | 62 138 | 62 149 | 62 159 | 62 170 | 62 180 | 62 190 | 62 201 | 62 211 |
| 419 | 62 221 | 62 232 | 62 242 | 62 252 | 62 263 | 62 273 | 62 284 | 62 294 | 62 304 | 62 315 |
| 420 | 62 325 | 62 335 | 62 346 | 62 356 | 62 366 | 62 377 | 62 387 | 62 397 | 62 408 | 62 418 |
| 421 | 62 428 | 62 439 | 62 449 | 62 459 | 62 469 | 62 480 | 62 490 | 62 500 | 62 511 | 62 521 |
| 422 | 62 531 | 62 542 | 62 552 | 62 562 | 62 572 | 62 583 | 62 593 | 62 603 | 62 613 | 62 624 |
| 423 | 62 634 | 62 644 | 62 655 | 62 665 | 62 675 | 62 685 | 62 696 | 62 706 | 62 716 | 62 726 |
| 424 | 62 737 | 62 747 | 62 757 | 62 767 | 62 778 | 62 788 | 62 798 | 62 808 | 62 818 | 62 829 |
| 425 | 62 839 | 62 849 | 62 859 | 62 870 | 62 880 | 62 890 | 62 900 | 62 910 | 62 921 | 62 931 |
| 426 | 62 941 | 62 951 | 62 961 | 62 972 | 62 982 | 62 992 | 63 002 | 63 012 | 63 022 | 63 033 |
| 427 | 63 043 | 63 053 | 63 063 | 63 073 | 63 083 | 63 094 | 63 104 | 63 114 | 63 124 | 63 134 |
| 428 | 63 144 | 63 155 | 63 165 | 63 175 | 63 185 | 63 195 | 63 205 | 63 215 | 63 225 | 63 236 |
| 429 | 63 246 | 63 256 | 63 266 | 63 276 | 63 286 | 63 296 | 63 306 | 63 317 | 63 327 | 63 337 |
| 430 | 63 347 | 63 357 | 63 367 | 63 377 | 63 387 | 63 397 | 63 407 | 63 417 | 63 428 | 63 438 |
| 431 | 63 448 | 63 458 | 63 468 | 63 478 | 63 488 | 63 498 | 63 508 | 63 518 | 63 528 | 63 538 |
| 432 | 63 548 | 63 558 | 63 568 | 64 579 | 63 589 | 63 599 | 63 609 | 63 619 | 63 629 | 63 639 |
| 433 | 63 649 | 63 659 | 63 669 | 63 679 | 63 689 | 63 699 | 63 709 | 63 719 | 63 729 | 63 739 |
| 434 | 63 749 | 63 759 | 63 769 | 63 779 | 63 789 | 63 799 | 63 809 | 63 819 | 63 829 | 63 839 |
| 435 | 63 849 | 63 859 | 63 869 | 63 879 | 63 889 | 63 899 | 63 909 | 63 919 | 63 929 | 63 939 |
| 436 | 63 949 | 63 959 | 63 969 | 63 979 | 63 988 | 63 998 | 64 008 | 64 018 | 64 028 | 64 038 |
| 437 | 64 048 | 64 058 | 64 068 | 64 078 | 64 088 | 64 098 | 64 108 | 64 118 | 64 128 | 64 137 |
| 438 | 64 147 | 64 157 | 64 167 | 64 177 | 64 187 | 64 197 | 64 207 | 64 217 | 64 227 | 64 237 |
| 439 | 64 246 | 64 256 | 64 266 | 64 276 | 64 286 | 64 296 | 64 306 | 64 316 | 64 326 | 64 335 |
| 440 | 64 345 | 64 355 | 64 365 | 64 375 | 64 385 | 64 395 | 64 404 | 64 414 | 64 424 | 64 434 |
| 441 | 64 444 | 64 454 | 64 464 | 64 473 | 64 483 | 64 493 | 64 503 | 64 513 | 64 523 | 64 532 |
| 442 | 64 542 | 64 552 | 64 562 | 64 572 | 64 582 | 64 591 | 64 601 | 64 611 | 64 621 | 64 631 |
| 443 | 64 640 | 64 650 | 64 660 | 64 670 | 64 680 | 64 689 | 64 699 | 64 709 | 64 719 | 64 729 |
| 444 | 64 738 | 64 748 | 64 758 | 64 768 | 64 777 | 64 787 | 64 797 | 64 807 | 64 816 | 64 826 |
| 445 | 64 836 | 64 846 | 64 856 | 64 865 | 64 875 | 64 885 | 64 895 | 64 904 | 64 914 | 64 924 |
| 446 | 64 933 | 64 943 | 64 953 | 64 963 | 64 972 | 64 982 | 64 992 | 65 002 | 65 011 | 65 021 |
| 447 | 65 031 | 65 040 | 65 050 | 65 060 | 65 070 | 65 079 | 65 089 | 65 099 | 65 108 | 65 118 |
| 448 | 65 128 | 65 137 | 65 147 | 65 157 | 65 167 | 65 176 | 65 186 | 65 196 | 65 205 | 65 215 |
| 449 | 65 225 | 65 234 | 65 244 | 65 254 | 65 263 | 65 273 | 65 283 | 65 292 | 65 302 | 65 312 |
| 450 | 65 321 | 65 331 | 65 341 | 65 350 | 65 360 | 65 369 | 65 379 | 65 389 | 65 398 | 65 408 |
| 451 | 65 418 | 65 427 | 65 437 | 65 447 | 65 456 | 65 466 | 65 475 | 65 485 | 65 495 | 65 504 |
| 452 | 65 514 | 65 523 | 65 533 | 65 543 | 65 552 | 65 562 | 65 571 | 65 581 | 65 591 | 65 600 |
| 453 | 65 610 | 65 619 | 65 629 | 65 639 | 65 648 | 65 658 | 65 667 | 65 677 | 65 686 | 65 696 |
| 454 | 65 706 | 65 715 | 65 725 | 65 734 | 65 744 | 65 753 | 65 763 | 65 772 | 65 782 | 65 792 |
| 455 | 65 801 | 65 811 | 65 820 | 65 830 | 65 839 | 65 849 | 65 858 | 65 868 | 65 877 | 65 887 |
| 456 | 65 896 | 65 906 | 65 916 | 65 925 | 65 935 | 65 944 | 65 954 | 65 963 | 65 973 | 65 982 |
| 457 | 65 992 | 66 001 | 66 011 | 66 020 | 66 030 | 66 039 | 66 049 | 66 058 | 66 068 | 66 077 |
| 458 | 66 087 | 66 096 | 66 106 | 66 115 | 66 124 | 66 134 | 66 143 | 66 153 | 66 162 | 66 172 |
| 459 | 66 181 | 66 191 | 66 200 | 66 210 | 66 219 | 66 229 | 66 238 | 66 247 | 66 257 | 66 266 |

COMMON LOGARITHMS OF NUMBERS

(Continued)

| N | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
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| 460 | 66 276 | 66 285 | 66 295 | 66 304 | 66 314 | 66 323 | 66 332 | 66 342 | 66 351 | 66 361 |
| 461 | 66 370 | 66 380 | 66 389 | 66 398 | 66 408 | 66 417 | 66 427 | 66 436 | 66 445 | 66 455 |
| 462 | 66 464 | 66 474 | 66 483 | 66 492 | 66 502 | 66 511 | 66 521 | 66 530 | 66 539 | 66 549 |
| 463 | 66 558 | 66 567 | 66 577 | 66 586 | 66 596 | 66 605 | 66 614 | 66 624 | 66 633 | 66 642 |
| 464 | 66 652 | 66 661 | 66 671 | 66 680 | 66 689 | 66 699 | 66 708 | 66 717 | 66 727 | 66 736 |
| 465 | 66 745 | 66 755 | 66 764 | 66 773 | 66 783 | 66 792 | 66 801 | 66 811 | 66 820 | 66 829 |
| 466 | 66 839 | 66 848 | 66 857 | 66 867 | 66 876 | 66 885 | 66 894 | 66 904 | 66 913 | 66 922 |
| 467 | 66 932 | 66 941 | 66 950 | 66 960 | 66 969 | 66 978 | 66 987 | 66 997 | 67 006 | 67 015 |
| 468 | 67 025 | 67 034 | 67 043 | 67 052 | 67 062 | 67 071 | 67 080 | 67 089 | 67 099 | 67 108 |
| 469 | 67 117 | 67 127 | 67 136 | 67 145 | 67 154 | 67 164 | 67 173 | 67 182 | 67 191 | 67 201 |
| 470 | 67 210 | 67 219 | 67 228 | 67 237 | 67 247 | 67 256 | 67 265 | 67 274 | 67 284 | 67 293 |
| 471 | 67 302 | 67 311 | 67 321 | 67 330 | 67 339 | 67 348 | 67 357 | 67 367 | 67 376 | 67 385 |
| 472 | 67 394 | 67 403 | 67 413 | 67 422 | 67 431 | 67 440 | 67 449 | 67 459 | 67 468 | 67 477 |
| 473 | 67 486 | 67 495 | 67 504 | 67 514 | 67 523 | 67 532 | 67 541 | 67 550 | 67 560 | 67 569 |
| 474 | 67 578 | 67 587 | 67 596 | 67 605 | 67 614 | 67 624 | 67 633 | 67 642 | 67 651 | 67 660 |
| 475 | 67 669 | 67 679 | 67 688 | 67 697 | 67 706 | 67 715 | 67 724 | 67 733 | 67 742 | 67 752 |
| 476 | 67 761 | 67 770 | 67 779 | 67 788 | 67 797 | 67 806 | 67 815 | 67 825 | 67 834 | 67 843 |
| 477 | 67 852 | 67 861 | 67 870 | 67 879 | 67 888 | 67 897 | 67 906 | 67 916 | 67 925 | 67 934 |
| 478 | 67 943 | 67 952 | 67 961 | 67 970 | 67 979 | 67 988 | 67 997 | 68 006 | 68 015 | 68 024 |
| 479 | 68 034 | 68 043 | 68 052 | 68 061 | 68 070 | 68 079 | 68 088 | 68 097 | 68 106 | 68 115 |
| 480 | 68 124 | 68 133 | 68 142 | 68 151 | 68 160 | 68 169 | 68 178 | 68 187 | 68 196 | 68 205 |
| 481 | 68 215 | 68 224 | 68 233 | 68 242 | 68 251 | 68 260 | 68 269 | 68 278 | 68 287 | 68 296 |
| 482 | 68 305 | 68 314 | 68 323 | 68 332 | 68 341 | 68 350 | 68 359 | 68 368 | 68 377 | 68 386 |
| 483 | 68 395 | 68 404 | 68 413 | 68 422 | 68 431 | 68 440 | 68 449 | 68 458 | 68 467 | 68 476 |
| 484 | 68 485 | 68 494 | 68 502 | 68 511 | 68 520 | 68 529 | 68 538 | 68 547 | 68 556 | 68 565 |
| 485 | 68 574 | 68 583 | 68 592 | 68 601 | 68 610 | 68 619 | 68 628 | 68 637 | 68 646 | 68 655 |
| 486 | 68 664 | 68 673 | 68 681 | 68 690 | 68 699 | 68 708 | 68 717 | 68 726 | 68 735 | 68 744 |
| 487 | 68 753 | 68 762 | 68 771 | 68 780 | 68 789 | 68 797 | 68 806 | 68 815 | 68 824 | 68 833 |
| 488 | 68 842 | 68 851 | 68 860 | 68 869 | 68 878 | 68 886 | 68 895 | 68 904 | 68 913 | 68 922 |
| 489 | 68 931 | 68 940 | 68 949 | 68 958 | 68 966 | 68 975 | 68 984 | 68 993 | 69 002 | 69 011 |
| 490 | 69 020 | 69 028 | 69 037 | 69 046 | 69 055 | 69 064 | 69 073 | 69 082 | 69 090 | 69 099 |
| 491 | 69 108 | 69 117 | 69 126 | 69 135 | 69 144 | 69 152 | 69 161 | 69 170 | 69 179 | 69 188 |
| 492 | 69 197 | 69 205 | 69 214 | 69 223 | 69 232 | 69 241 | 69 249 | 69 258 | 69 267 | 69 276 |
| 493 | 69 285 | 69 294 | 69 302 | 69 311 | 69 320 | 69 329 | 69 338 | 69 346 | 69 355 | 69 364 |
| 494 | 69 373 | 69 381 | 69 390 | 69 399 | 69 408 | 69 417 | 69 425 | 69 434 | 69 443 | 69 452 |
| 495 | 69 461 | 69 469 | 69 478 | 69 487 | 69 496 | 69 504 | 69 513 | 69 522 | 69 531 | 69 539 |
| 496 | 69 548 | 69 557 | 69 566 | 69 574 | 69 583 | 69 592 | 69 601 | 69 609 | 69 618 | 69 627 |
| 497 | 69 636 | 69 644 | 69 653 | 69 662 | 69 671 | 69 679 | 69 688 | 69 697 | 69 705 | 69 714 |
| 498 | 69 723 | 69 732 | 69 740 | 69 749 | 69 758 | 69 767 | 69 775 | 69 784 | 69 793 | 69 801 |
| 499 | 69 810 | 69 819 | 69 827 | 69 836 | 69 845 | 69 854 | 69 862 | 69 871 | 69 880 | 69 888 |
| 500 | 69 897 | 69 906 | 69 914 | 69 923 | 69 932 | 69 940 | 69 949 | 69 958 | 69 966 | 69 975 |
| 501 | 69 984 | 69 992 | 70 001 | 70 010 | 70 018 | 70 027 | 70 036 | 70 044 | 70 053 | 70 062 |
| 502 | 70 070 | 70 079 | 70 088 | 70 096 | 70 105 | 70 114 | 70 122 | 70 131 | 70 140 | 70 148 |
| 503 | 70 157 | 70 165 | 70 174 | 70 183 | 70 191 | 70 200 | 70 209 | 70 217 | 70 226 | 70 234 |
| 504 | 70 243 | 70 252 | 70 260 | 70 269 | 70 278 | 70 286 | 70 295 | 70 303 | 70 312 | 70 321 |

COMMON LOGARITHMS OF NUMBERS

(Continued)

| N | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|-----|--------|--------|--------|--------|--------|--------|--------|--------|--------|--------|
| 505 | 70 329 | 70 338 | 70 346 | 70 355 | 70 364 | 70 372 | 70 381 | 70 389 | 70 398 | 70 406 |
| 506 | 70 415 | 70 424 | 70 432 | 70 441 | 70 449 | 70 458 | 70 467 | 70 475 | 70 484 | 70 492 |
| 507 | 70 501 | 70 509 | 70 518 | 70 526 | 70 535 | 70 544 | 70 552 | 70 561 | 70 569 | 70 578 |
| 508 | 70 586 | 70 595 | 70 603 | 70 612 | 70 621 | 70 629 | 70 638 | 70 646 | 70 655 | 70 663 |
| 509 | 70 672 | 70 680 | 70 689 | 70 697 | 70 706 | 70 714 | 70 723 | 70 731 | 70 740 | 70 749 |
| 510 | 70 757 | 70 766 | 70 774 | 70 783 | 70 791 | 70 800 | 70 808 | 70 817 | 70 825 | 70 834 |
| 511 | 70 842 | 70 851 | 70 859 | 70 868 | 70 876 | 70 885 | 70 893 | 70 902 | 70 910 | 70 919 |
| 512 | 70 927 | 70 935 | 70 944 | 70 952 | 70 961 | 70 969 | 70 978 | 70 986 | 70 995 | 71 003 |
| 513 | 71 012 | 71 020 | 71 029 | 71 037 | 71 046 | 71 054 | 71 063 | 71 071 | 71 079 | 71 088 |
| 514 | 71 096 | 71 105 | 71 113 | 71 122 | 71 130 | 71 139 | 71 147 | 71 155 | 71 164 | 71 172 |
| 515 | 71 181 | 71 189 | 71 198 | 71 206 | 71 214 | 71 223 | 71 231 | 71 240 | 71 248 | 71 257 |
| 516 | 71 265 | 71 273 | 71 282 | 71 290 | 71 299 | 71 307 | 71 315 | 71 324 | 71 332 | 71 341 |
| 517 | 71 349 | 71 357 | 71 366 | 71 374 | 71 383 | 71 391 | 71 399 | 71 408 | 71 416 | 71 425 |
| 518 | 71 433 | 71 441 | 71 450 | 71 458 | 71 466 | 71 475 | 71 483 | 71 492 | 71 500 | 71 508 |
| 519 | 71 517 | 71 525 | 71 533 | 71 542 | 71 550 | 71 559 | 71 567 | 71 575 | 71 584 | 71 592 |
| 520 | 71 600 | 71 609 | 71 617 | 71 625 | 71 634 | 71 642 | 71 650 | 71 659 | 71 667 | 71 675 |
| 521 | 71 684 | 71 692 | 71 700 | 71 709 | 71 717 | 71 725 | 71 734 | 71 742 | 71 750 | 71 759 |
| 522 | 71 767 | 71 775 | 71 784 | 71 792 | 71 800 | 71 809 | 71 817 | 71 825 | 71 834 | 71 842 |
| 523 | 71 850 | 71 858 | 71 867 | 71 875 | 71 883 | 71 892 | 71 900 | 71 908 | 71 917 | 71 925 |
| 524 | 71 933 | 71 941 | 71 950 | 71 958 | 71 966 | 71 975 | 71 983 | 71 991 | 71 999 | 72 008 |
| 525 | 72 016 | 72 024 | 72 032 | 72 041 | 72 049 | 72 057 | 72 066 | 72 074 | 72 082 | 72 090 |
| 526 | 72 099 | 72 107 | 72 115 | 72 123 | 72 132 | 72 140 | 72 148 | 72 156 | 72 165 | 72 173 |
| 527 | 72 181 | 72 189 | 72 198 | 72 206 | 72 214 | 72 222 | 72 230 | 72 239 | 72 247 | 72 255 |
| 528 | 72 263 | 72 272 | 72 280 | 72 288 | 72 296 | 72 304 | 72 313 | 72 321 | 72 329 | 72 337 |
| 529 | 72 346 | 72 354 | 72 362 | 72 370 | 72 378 | 72 387 | 72 395 | 72 403 | 72 411 | 72 419 |
| 530 | 72 428 | 72 436 | 72 444 | 72 452 | 72 460 | 72 469 | 72 477 | 72 485 | 72 493 | 72 501 |
| 531 | 72 509 | 72 518 | 72 526 | 72 534 | 72 542 | 72 550 | 72 558 | 72 567 | 72 575 | 72 583 |
| 532 | 72 591 | 72 599 | 72 607 | 72 616 | 72 624 | 72 632 | 72 640 | 72 648 | 72 656 | 72 665 |
| 533 | 72 673 | 72 681 | 72 689 | 72 697 | 72 705 | 72 713 | 72 722 | 72 730 | 72 738 | 72 746 |
| 534 | 72 754 | 72 762 | 72 770 | 72 779 | 72 787 | 72 795 | 72 803 | 72 811 | 72 819 | 72 827 |
| 535 | 72 835 | 72 843 | 72 852 | 72 860 | 72 868 | 72 876 | 72 884 | 72 892 | 72 900 | 72 908 |
| 536 | 72 916 | 72 925 | 72 933 | 72 941 | 72 949 | 72 957 | 72 965 | 72 973 | 72 981 | 72 989 |
| 537 | 72 997 | 73 006 | 73 014 | 73 022 | 73 030 | 73 038 | 73 046 | 73 054 | 73 062 | 73 070 |
| 538 | 73 078 | 73 086 | 73 094 | 73 102 | 73 111 | 73 119 | 73 127 | 73 135 | 73 143 | 73 151 |
| 539 | 73 159 | 73 167 | 73 175 | 73 183 | 73 191 | 73 199 | 73 207 | 73 215 | 73 223 | 73 231 |
| 540 | 73 239 | 73 247 | 73 255 | 73 263 | 73 272 | 73 280 | 73 288 | 73 296 | 73 304 | 73 312 |
| 541 | 73 320 | 73 328 | 73 336 | 73 344 | 73 352 | 73 360 | 73 368 | 73 376 | 73 384 | 73 392 |
| 542 | 73 400 | 73 408 | 73 416 | 73 424 | 73 432 | 73 440 | 73 448 | 73 456 | 73 464 | 73 472 |
| 543 | 73 480 | 73 488 | 73 496 | 73 504 | 73 512 | 73 520 | 73 528 | 73 536 | 73 544 | 73 552 |
| 544 | 73 560 | 73 568 | 73 576 | 73 584 | 73 592 | 73 600 | 73 608 | 73 616 | 73 624 | 73 632 |
| 545 | 73 640 | 73 648 | 73 656 | 73 664 | 73 672 | 73 679 | 73 687 | 73 695 | 73 703 | 73 711 |
| 546 | 73 719 | 73 727 | 73 735 | 73 743 | 73 751 | 73 759 | 73 767 | 73 775 | 73 783 | 73 791 |
| 547 | 73 799 | 73 807 | 73 815 | 73 823 | 73 830 | 73 838 | 73 846 | 73 854 | 73 862 | 73 870 |
| 548 | 73 878 | 73 886 | 73 894 | 73 902 | 73 910 | 73 918 | 73 926 | 73 933 | 73 941 | 73 949 |
| 549 | 73 957 | 73 965 | 73 973 | 73 981 | 73 989 | 73 997 | 74 005 | 74 013 | 74 020 | 74 028 |

COMMON LOGARITHMS OF NUMBERS

(Continued)

| <i>N</i> | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|----------|--------|--------|--------|--------|--------|--------|--------|--------|--------|--------|
| 550 | 74 036 | 74 044 | 74 052 | 74 060 | 74 068 | 74 076 | 74 084 | 74 092 | 74 099 | 74 107 |
| 551 | 74 115 | 74 123 | 74 131 | 74 139 | 74 147 | 74 155 | 74 162 | 74 170 | 74 178 | 74 186 |
| 552 | 74 194 | 74 202 | 74 210 | 74 218 | 74 225 | 74 233 | 74 241 | 74 249 | 74 257 | 74 265 |
| 553 | 74 273 | 74 280 | 74 288 | 74 296 | 74 304 | 74 312 | 74 320 | 74 327 | 74 335 | 74 343 |
| 554 | 74 351 | 74 359 | 74 367 | 74 374 | 74 382 | 74 390 | 74 398 | 74 406 | 74 414 | 74 421 |
| 555 | 74 429 | 74 437 | 74 445 | 74 453 | 74 461 | 74 468 | 74 476 | 74 484 | 74 492 | 74 500 |
| 556 | 74 507 | 74 515 | 74 523 | 74 531 | 74 539 | 74 547 | 74 554 | 74 562 | 74 570 | 74 578 |
| 557 | 74 586 | 74 593 | 74 601 | 74 609 | 74 617 | 74 624 | 74 632 | 74 640 | 74 648 | 74 656 |
| 558 | 74 663 | 74 671 | 74 679 | 74 687 | 74 695 | 74 702 | 74 710 | 74 718 | 74 726 | 74 733 |
| 559 | 74 741 | 74 749 | 74 757 | 74 764 | 74 772 | 74 780 | 74 788 | 74 796 | 74 803 | 74 811 |
| 560 | 74 819 | 74 827 | 74 834 | 74 842 | 74 850 | 74 858 | 74 865 | 74 873 | 74 881 | 74 889 |
| 561 | 74 896 | 74 904 | 74 912 | 74 920 | 74 927 | 74 935 | 74 943 | 74 950 | 74 958 | 74 966 |
| 562 | 74 974 | 74 981 | 74 989 | 74 997 | 75 005 | 75 012 | 75 020 | 75 028 | 75 035 | 75 043 |
| 563 | 75 051 | 75 059 | 75 066 | 75 074 | 75 082 | 75 089 | 75 097 | 75 105 | 75 113 | 75 120 |
| 564 | 75 128 | 75 136 | 75 143 | 75 151 | 75 159 | 75 166 | 75 174 | 75 182 | 75 189 | 75 197 |
| 565 | 75 205 | 75 213 | 75 220 | 75 228 | 75 236 | 75 243 | 75 251 | 75 259 | 75 266 | 75 274 |
| 566 | 75 282 | 75 289 | 75 297 | 75 305 | 75 312 | 75 320 | 75 328 | 75 335 | 75 343 | 75 351 |
| 567 | 75 358 | 75 366 | 75 374 | 75 381 | 75 389 | 75 397 | 75 404 | 75 412 | 75 420 | 75 427 |
| 568 | 75 435 | 75 442 | 75 450 | 75 458 | 75 465 | 75 473 | 75 481 | 75 488 | 75 496 | 75 504 |
| 569 | 75 511 | 75 519 | 75 526 | 75 534 | 75 542 | 75 549 | 75 557 | 75 565 | 75 572 | 75 580 |
| 570 | 75 587 | 75 595 | 75 603 | 75 610 | 75 618 | 75 626 | 75 633 | 75 641 | 75 648 | 75 656 |
| 571 | 75 664 | 75 671 | 75 679 | 75 686 | 75 694 | 75 702 | 75 709 | 75 717 | 75 724 | 75 732 |
| 572 | 75 740 | 75 747 | 75 755 | 75 762 | 75 770 | 75 778 | 75 785 | 75 793 | 75 800 | 75 808 |
| 573 | 75 815 | 75 823 | 75 831 | 75 838 | 75 846 | 75 853 | 75 861 | 75 868 | 75 876 | 75 884 |
| 574 | 75 891 | 75 899 | 75 906 | 75 914 | 75 921 | 75 929 | 75 937 | 75 944 | 75 952 | 75 959 |
| 575 | 75 967 | 75 974 | 75 982 | 75 989 | 75 997 | 76 005 | 76 012 | 76 020 | 76 027 | 76 035 |
| 576 | 76 042 | 76 050 | 76 057 | 76 065 | 76 072 | 76 080 | 76 087 | 76 095 | 76 103 | 76 110 |
| 577 | 76 118 | 76 125 | 76 133 | 76 140 | 76 148 | 76 155 | 76 163 | 76 170 | 76 178 | 76 185 |
| 578 | 76 193 | 76 200 | 76 208 | 76 215 | 76 223 | 76 230 | 76 238 | 76 245 | 76 253 | 76 260 |
| 579 | 76 268 | 76 275 | 76 283 | 76 290 | 76 298 | 76 305 | 76 313 | 76 320 | 76 328 | 76 335 |
| 580 | 76 343 | 76 350 | 76 358 | 76 365 | 76 373 | 76 380 | 76 388 | 76 395 | 76 403 | 76 410 |
| 581 | 76 418 | 76 425 | 76 433 | 76 440 | 76 448 | 76 455 | 76 462 | 76 470 | 76 477 | 76 485 |
| 582 | 76 492 | 76 500 | 76 507 | 76 515 | 76 522 | 76 530 | 76 537 | 76 545 | 76 552 | 76 559 |
| 583 | 76 567 | 76 574 | 76 582 | 76 589 | 76 597 | 76 604 | 76 612 | 76 619 | 76 626 | 76 634 |
| 584 | 76 641 | 76 649 | 76 656 | 76 664 | 76 671 | 76 678 | 76 686 | 76 693 | 76 701 | 76 708 |
| 585 | 76 716 | 76 723 | 76 730 | 76 738 | 76 745 | 76 753 | 76 760 | 76 768 | 76 775 | 76 782 |
| 586 | 76 790 | 76 797 | 76 805 | 76 812 | 76 819 | 76 827 | 76 834 | 76 842 | 76 849 | 76 856 |
| 587 | 76 864 | 76 871 | 76 879 | 76 886 | 76 893 | 76 901 | 76 908 | 76 916 | 76 923 | 76 930 |
| 588 | 76 938 | 76 945 | 76 953 | 76 960 | 76 967 | 76 975 | 76 982 | 76 989 | 76 997 | 77 004 |
| 589 | 77 012 | 77 019 | 77 026 | 77 034 | 77 041 | 77 048 | 77 056 | 77 063 | 77 070 | 77 078 |
| 590 | 77 085 | 77 093 | 77 100 | 77 107 | 77 115 | 77 122 | 77 129 | 77 137 | 77 144 | 77 151 |
| 591 | 77 159 | 77 166 | 77 173 | 77 181 | 77 188 | 77 195 | 77 203 | 77 210 | 77 217 | 77 225 |
| 592 | 77 232 | 77 240 | 77 247 | 77 254 | 77 262 | 77 269 | 77 276 | 77 283 | 77 291 | 77 298 |
| 593 | 77 305 | 77 313 | 77 320 | 77 327 | 77 335 | 77 342 | 77 349 | 77 357 | 77 364 | 77 371 |
| 594 | 77 379 | 77 386 | 77 393 | 77 401 | 77 408 | 77 415 | 77 422 | 77 430 | 77 437 | 77 444 |

COMMON LOGARITHMS OF NUMBERS

(Continued)

| <i>N</i> | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|----------|--------|--------|--------|--------|--------|--------|--------|--------|--------|--------|
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| 596 | 77 525 | 77 532 | 77 539 | 77 546 | 77 554 | 77 561 | 77 568 | 77 576 | 77 583 | 77 590 |
| 597 | 77 597 | 77 605 | 77 612 | 77 619 | 77 627 | 77 634 | 77 641 | 77 648 | 77 656 | 77 663 |
| 598 | 77 670 | 77 677 | 77 685 | 77 692 | 77 699 | 77 706 | 77 714 | 77 721 | 77 728 | 77 735 |
| 599 | 77 743 | 77 750 | 77 757 | 77 764 | 77 772 | 77 779 | 77 786 | 77 793 | 77 801 | 77 808 |
| 600 | 77 815 | 77 822 | 77 830 | 77 837 | 77 844 | 77 851 | 77 859 | 77 866 | 77 873 | 77 880 |
| 601 | 77 887 | 77 895 | 77 902 | 77 909 | 77 916 | 77 924 | 77 931 | 77 938 | 77 945 | 77 952 |
| 602 | 77 960 | 77 967 | 77 974 | 77 981 | 77 988 | 77 996 | 78 003 | 78 010 | 78 017 | 78 025 |
| 603 | 78 032 | 78 039 | 78 046 | 78 053 | 78 061 | 78 068 | 78 075 | 78 082 | 78 089 | 78 097 |
| 604 | 78 104 | 78 111 | 78 118 | 78 125 | 78 132 | 78 140 | 78 147 | 78 154 | 78 161 | 78 168 |
| 605 | 78 176 | 78 183 | 78 190 | 78 197 | 78 204 | 78 211 | 78 219 | 78 226 | 78 233 | 78 240 |
| 606 | 78 247 | 78 254 | 78 262 | 78 269 | 78 276 | 78 283 | 78 290 | 78 297 | 78 305 | 78 312 |
| 607 | 78 319 | 78 326 | 78 333 | 78 340 | 78 347 | 78 355 | 78 362 | 78 369 | 78 376 | 78 383 |
| 608 | 78 390 | 78 398 | 78 405 | 78 412 | 78 419 | 78 426 | 78 433 | 78 440 | 78 447 | 78 455 |
| 609 | 78 462 | 78 469 | 78 476 | 78 483 | 78 490 | 78 497 | 78 504 | 78 512 | 78 519 | 78 526 |
| 610 | 78 533 | 78 540 | 78 547 | 78 554 | 78 561 | 78 569 | 78 576 | 78 583 | 78 590 | 78 597 |
| 611 | 78 604 | 78 611 | 78 618 | 78 625 | 78 633 | 78 640 | 78 647 | 78 654 | 78 661 | 78 668 |
| 612 | 78 675 | 78 682 | 78 689 | 78 696 | 78 704 | 78 711 | 78 718 | 78 725 | 78 732 | 78 739 |
| 613 | 78 746 | 78 753 | 78 760 | 78 767 | 78 774 | 78 781 | 78 789 | 78 796 | 78 803 | 78 810 |
| 614 | 78 817 | 78 824 | 78 831 | 78 838 | 78 845 | 78 852 | 78 859 | 78 866 | 78 873 | 78 880 |
| 615 | 78 888 | 78 895 | 78 902 | 78 909 | 78 916 | 78 923 | 78 930 | 78 937 | 78 944 | 78 951 |
| 616 | 78 958 | 78 965 | 78 972 | 78 979 | 78 986 | 78 993 | 79 000 | 79 007 | 79 014 | 79 021 |
| 617 | 79 029 | 79 036 | 79 043 | 79 050 | 79 057 | 79 064 | 79 071 | 79 078 | 79 085 | 79 092 |
| 618 | 79 099 | 79 106 | 79 113 | 79 120 | 79 127 | 79 134 | 79 141 | 79 148 | 79 155 | 79 162 |
| 619 | 79 169 | 79 176 | 79 183 | 79 190 | 79 197 | 79 204 | 79 211 | 79 218 | 79 225 | 79 232 |
| 620 | 79 239 | 79 246 | 79 253 | 79 260 | 79 267 | 79 274 | 79 281 | 79 288 | 79 295 | 79 302 |
| 621 | 79 309 | 79 316 | 79 323 | 79 330 | 79 337 | 79 344 | 79 351 | 79 358 | 79 365 | 79 372 |
| 622 | 79 379 | 79 386 | 79 393 | 79 400 | 79 407 | 79 414 | 79 421 | 79 428 | 79 435 | 79 442 |
| 623 | 79 449 | 79 456 | 79 463 | 79 470 | 79 477 | 79 484 | 79 491 | 79 498 | 79 505 | 79 511 |
| 624 | 79 518 | 79 525 | 79 532 | 79 539 | 79 546 | 79 553 | 79 560 | 79 567 | 79 574 | 79 581 |
| 625 | 79 588 | 79 595 | 79 602 | 79 609 | 79 616 | 79 623 | 79 630 | 79 637 | 79 644 | 79 650 |
| 626 | 79 657 | 79 664 | 79 671 | 79 678 | 79 685 | 79 692 | 79 699 | 79 706 | 79 713 | 79 720 |
| 627 | 79 727 | 79 734 | 79 741 | 79 748 | 79 754 | 79 761 | 79 768 | 79 775 | 79 782 | 79 789 |
| 628 | 79 796 | 79 803 | 79 810 | 79 817 | 79 824 | 79 831 | 79 837 | 79 844 | 79 851 | 79 858 |
| 629 | 79 865 | 79 872 | 79 879 | 79 886 | 79 893 | 79 900 | 79 906 | 79 913 | 79 920 | 79 927 |
| 630 | 79 934 | 79 941 | 79 948 | 79 955 | 79 962 | 79 969 | 79 975 | 79 982 | 79 989 | 79 996 |
| 631 | 80 003 | 80 010 | 80 017 | 80 024 | 80 030 | 80 037 | 80 044 | 80 051 | 80 058 | 80 065 |
| 632 | 80 072 | 80 079 | 80 085 | 80 092 | 80 099 | 80 106 | 80 113 | 80 120 | 80 127 | 80 134 |
| 633 | 80 140 | 80 147 | 80 154 | 80 161 | 80 168 | 80 175 | 80 182 | 80 188 | 80 195 | 80 202 |
| 634 | 80 209 | 80 216 | 80 223 | 80 229 | 80 236 | 80 243 | 80 250 | 80 257 | 80 264 | 80 271 |
| 635 | 80 277 | 80 284 | 80 291 | 80 298 | 80 305 | 80 312 | 80 318 | 80 325 | 80 332 | 80 339 |
| 636 | 80 346 | 80 353 | 80 359 | 80 366 | 80 373 | 80 380 | 80 387 | 80 393 | 80 400 | 80 407 |
| 637 | 80 414 | 80 421 | 80 428 | 80 434 | 80 441 | 80 448 | 80 455 | 80 462 | 80 468 | 80 475 |
| 638 | 80 482 | 80 489 | 80 496 | 80 502 | 80 509 | 80 516 | 80 523 | 80 530 | 80 536 | 80 543 |
| 639 | 80 550 | 80 557 | 80 564 | 80 570 | 80 577 | 80 584 | 80 591 | 80 598 | 80 604 | 80 611 |

COMMON LOGARITHMS OF NUMBERS

(Continued)

| N | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
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| 641 | 80 686 | 80 693 | 80 699 | 80 706 | 80 713 | 80 720 | 80 726 | 80 733 | 80 740 | 80 747 |
| 642 | 80 754 | 80 760 | 80 767 | 80 774 | 80 781 | 80 787 | 80 794 | 80 801 | 80 808 | 80 814 |
| 643 | 80 821 | 80 828 | 80 835 | 80 841 | 80 848 | 80 855 | 80 862 | 80 868 | 80 875 | 80 882 |
| 644 | 80 889 | 80 895 | 80 902 | 80 909 | 80 916 | 80 922 | 80 929 | 80 936 | 80 943 | 80 949 |
| 645 | 80 956 | 80 963 | 80 969 | 80 976 | 80 983 | 80 990 | 80 996 | 81 003 | 81 010 | 81 017 |
| 646 | 81 023 | 81 030 | 81 037 | 81 043 | 81 050 | 81 057 | 81 064 | 81 070 | 81 077 | 81 084 |
| 647 | 81 090 | 81 097 | 81 104 | 81 111 | 81 117 | 81 124 | 81 131 | 81 137 | 81 144 | 81 151 |
| 648 | 81 158 | 81 164 | 81 171 | 81 178 | 81 184 | 81 191 | 81 198 | 81 204 | 81 211 | 81 218 |
| 649 | 81 224 | 81 231 | 81 238 | 81 245 | 81 251 | 81 258 | 81 265 | 81 271 | 81 278 | 81 285 |
| 650 | 81 291 | 81 298 | 81 305 | 81 311 | 81 318 | 81 325 | 81 331 | 81 338 | 81 345 | 81 351 |
| 651 | 81 358 | 81 365 | 81 371 | 81 378 | 81 385 | 81 391 | 81 398 | 81 405 | 81 411 | 81 418 |
| 652 | 81 425 | 81 431 | 81 438 | 81 445 | 81 451 | 81 458 | 81 465 | 81 471 | 81 478 | 81 485 |
| 653 | 81 491 | 81 498 | 81 505 | 81 511 | 81 518 | 81 525 | 81 531 | 81 538 | 81 544 | 81 551 |
| 654 | 81 558 | 81 564 | 81 571 | 81 578 | 81 584 | 81 591 | 81 598 | 81 604 | 81 611 | 81 617 |
| 655 | 81 624 | 81 631 | 81 637 | 81 644 | 81 651 | 81 657 | 81 664 | 81 671 | 81 677 | 81 684 |
| 656 | 81 690 | 81 697 | 81 704 | 81 710 | 81 717 | 81 723 | 81 730 | 81 737 | 81 743 | 81 750 |
| 657 | 81 757 | 81 763 | 81 770 | 81 776 | 81 783 | 81 790 | 81 796 | 81 803 | 81 809 | 81 816 |
| 658 | 81 823 | 81 829 | 81 836 | 81 842 | 81 849 | 81 856 | 81 862 | 81 869 | 81 875 | 81 882 |
| 659 | 81 889 | 81 895 | 81 902 | 81 908 | 81 915 | 81 921 | 81 928 | 81 935 | 81 941 | 81 948 |
| 660 | 81 954 | 81 961 | 81 968 | 81 974 | 81 981 | 81 987 | 81 994 | 82 000 | 82 007 | 82 014 |
| 661 | 82 020 | 82 027 | 82 033 | 82 040 | 82 046 | 82 053 | 82 060 | 82 066 | 82 073 | 82 079 |
| 662 | 82 086 | 82 092 | 82 099 | 82 105 | 82 112 | 82 119 | 82 125 | 82 132 | 82 138 | 82 145 |
| 663 | 82 151 | 82 158 | 82 164 | 82 171 | 82 178 | 82 184 | 82 191 | 82 197 | 82 204 | 82 210 |
| 664 | 82 217 | 82 223 | 82 230 | 82 236 | 82 243 | 82 249 | 82 256 | 82 263 | 82 269 | 82 276 |
| 665 | 82 282 | 82 289 | 82 295 | 82 302 | 82 308 | 82 315 | 82 321 | 82 328 | 82 334 | 82 341 |
| 666 | 82 347 | 82 354 | 82 360 | 82 367 | 82 373 | 82 380 | 82 387 | 82 393 | 82 400 | 82 406 |
| 667 | 82 413 | 82 419 | 82 426 | 82 432 | 82 439 | 82 445 | 82 452 | 82 458 | 82 465 | 82 471 |
| 668 | 82 478 | 82 484 | 82 491 | 82 497 | 82 504 | 82 510 | 82 517 | 82 523 | 82 530 | 82 536 |
| 669 | 82 543 | 82 549 | 82 556 | 82 562 | 82 569 | 82 575 | 82 582 | 82 588 | 82 595 | 82 601 |
| 670 | 82 607 | 82 614 | 82 620 | 82 627 | 82 633 | 82 640 | 82 646 | 82 653 | 82 659 | 82 666 |
| 671 | 82 672 | 82 679 | 82 685 | 82 692 | 82 698 | 82 705 | 82 711 | 82 718 | 82 724 | 82 730 |
| 672 | 82 737 | 82 743 | 82 750 | 82 756 | 82 763 | 82 769 | 82 776 | 82 782 | 82 789 | 82 795 |
| 673 | 82 802 | 82 808 | 82 814 | 82 821 | 82 827 | 82 834 | 82 840 | 82 847 | 82 853 | 82 860 |
| 674 | 82 866 | 82 872 | 82 879 | 82 885 | 82 892 | 82 898 | 82 905 | 82 911 | 82 918 | 82 924 |
| 675 | 82 930 | 82 937 | 82 943 | 82 950 | 82 956 | 82 963 | 82 969 | 82 975 | 82 982 | 82 988 |
| 676 | 82 995 | 83 001 | 83 008 | 83 014 | 83 020 | 83 027 | 83 033 | 83 040 | 83 046 | 83 052 |
| 677 | 83 059 | 83 065 | 83 072 | 83 078 | 83 085 | 83 091 | 83 097 | 83 104 | 83 110 | 83 117 |
| 678 | 83 123 | 83 129 | 83 136 | 83 142 | 83 149 | 83 155 | 83 161 | 83 168 | 83 174 | 83 181 |
| 679 | 83 187 | 83 193 | 83 200 | 83 206 | 83 213 | 83 219 | 83 225 | 83 232 | 83 238 | 83 245 |
| 680 | 83 251 | 83 257 | 83 264 | 83 270 | 83 276 | 83 283 | 83 289 | 83 296 | 83 302 | 83 308 |
| 681 | 83 315 | 83 321 | 83 327 | 83 334 | 83 340 | 83 347 | 83 353 | 83 359 | 83 366 | 83 372 |
| 682 | 83 378 | 83 385 | 83 391 | 83 398 | 83 404 | 83 410 | 83 417 | 83 423 | 83 429 | 83 436 |
| 683 | 83 442 | 83 448 | 83 455 | 83 461 | 83 467 | 83 474 | 83 480 | 83 487 | 83 493 | 83 499 |
| 684 | 83 506 | 83 512 | 83 518 | 83 525 | 83 531 | 83 537 | 83 544 | 83 550 | 83 556 | 83 563 |

COMMON LOGARITHMS OF NUMBERS

(Continued)

| <i>N</i> | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|----------|--------|--------|--------|--------|--------|--------|--------|--------|--------|--------|
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| 686 | 83 632 | 83 639 | 83 645 | 83 651 | 83 658 | 83 664 | 83 670 | 83 677 | 83 683 | 83 689 |
| 687 | 83 696 | 83 702 | 83 708 | 83 715 | 83 721 | 83 727 | 83 734 | 83 740 | 83 746 | 83 753 |
| 688 | 83 759 | 83 765 | 83 771 | 83 778 | 83 784 | 83 790 | 83 797 | 83 803 | 83 809 | 83 816 |
| 689 | 83 822 | 83 828 | 83 835 | 83 841 | 83 847 | 83 853 | 83 860 | 83 866 | 83 872 | 83 879 |
| 690 | 83 885 | 83 891 | 83 897 | 83 904 | 83 910 | 83 916 | 83 923 | 83 929 | 83 935 | 83 942 |
| 691 | 83 948 | 83 954 | 83 960 | 83 967 | 83 973 | 83 979 | 83 985 | 83 992 | 83 998 | 84 004 |
| 692 | 84 011 | 84 017 | 84 023 | 84 029 | 84 036 | 84 042 | 84 048 | 84 055 | 84 061 | 84 067 |
| 693 | 84 073 | 84 080 | 84 086 | 84 092 | 84 098 | 84 105 | 84 111 | 84 117 | 84 123 | 84 130 |
| 694 | 84 136 | 84 142 | 84 148 | 84 155 | 84 161 | 84 167 | 84 173 | 84 180 | 84 186 | 84 192 |
| 695 | 84 198 | 84 205 | 84 211 | 84 217 | 84 223 | 84 230 | 84 236 | 84 242 | 84 248 | 84 255 |
| 696 | 84 261 | 84 267 | 84 273 | 84 280 | 84 286 | 84 292 | 84 298 | 84 305 | 84 311 | 84 317 |
| 697 | 84 323 | 84 330 | 84 336 | 84 342 | 84 348 | 84 354 | 84 361 | 84 367 | 84 373 | 84 379 |
| 698 | 84 386 | 84 392 | 84 398 | 84 404 | 84 410 | 84 417 | 84 423 | 84 429 | 84 435 | 84 442 |
| 699 | 84 448 | 84 454 | 84 460 | 84 466 | 84 473 | 84 479 | 84 485 | 48 491 | 84 497 | 84 504 |
| 700 | 84 510 | 84 516 | 84 522 | 84 528 | 84 535 | 84 541 | 84 547 | 84 553 | 84 559 | 84 566 |
| 701 | 84 572 | 84 578 | 84 584 | 84 590 | 84 597 | 84 603 | 84 609 | 84 615 | 84 621 | 84 628 |
| 702 | 84 634 | 84 640 | 84 646 | 84 652 | 84 658 | 84 665 | 84 671 | 84 677 | 84 683 | 84 689 |
| 703 | 84 696 | 84 702 | 84 708 | 84 714 | 84 720 | 84 726 | 84 733 | 84 739 | 84 745 | 84 751 |
| 704 | 84 757 | 84 763 | 84 770 | 84 776 | 84 782 | 84 788 | 84 794 | 84 800 | 84 807 | 84 813 |
| 705 | 84 819 | 84 825 | 84 831 | 84 837 | 84 844 | 84 850 | 84 856 | 84 862 | 84 868 | 84 874 |
| 706 | 84 880 | 84 887 | 84 893 | 84 899 | 84 905 | 84 911 | 84 917 | 84 924 | 84 930 | 84 936 |
| 707 | 84 942 | 84 948 | 84 954 | 84 960 | 84 967 | 84 973 | 84 979 | 84 985 | 84 991 | 84 997 |
| 708 | 85 003 | 85 009 | 85 016 | 85 022 | 85 028 | 85 034 | 85 040 | 85 046 | 85 052 | 85 058 |
| 709 | 85 065 | 85 071 | 85 077 | 85 083 | 85 089 | 85 095 | 85 101 | 85 107 | 85 114 | 85 120 |
| 710 | 85 126 | 85 132 | 85 138 | 85 144 | 85 150 | 85 156 | 85 163 | 85 169 | 85 175 | 85 181 |
| 711 | 85 187 | 85 193 | 85 199 | 85 205 | 85 211 | 85 217 | 85 224 | 85 230 | 85 236 | 85 242 |
| 712 | 85 248 | 85 254 | 85 260 | 85 266 | 85 272 | 85 278 | 85 285 | 85 291 | 85 297 | 85 303 |
| 713 | 85 309 | 85 315 | 85 321 | 85 327 | 85 333 | 85 339 | 85 345 | 85 352 | 85 358 | 85 364 |
| 714 | 85 370 | 85 376 | 85 382 | 85 388 | 85 394 | 85 400 | 85 406 | 85 412 | 85 418 | 85 425 |
| 715 | 85 431 | 85 437 | 85 443 | 85 449 | 85 455 | 85 461 | 85 467 | 85 473 | 85 479 | 85 485 |
| 716 | 85 491 | 85 497 | 85 503 | 85 509 | 85 516 | 85 522 | 85 528 | 85 534 | 85 540 | 85 546 |
| 717 | 85 552 | 85 558 | 85 564 | 85 570 | 85 576 | 85 582 | 85 588 | 85 594 | 85 600 | 85 606 |
| 718 | 85 612 | 85 618 | 85 625 | 85 631 | 85 637 | 85 643 | 85 649 | 85 655 | 85 661 | 85 667 |
| 719 | 85 673 | 85 679 | 85 685 | 85 691 | 85 697 | 85 703 | 85 709 | 85 715 | 85 721 | 85 727 |
| 720 | 85 733 | 85 739 | 85 745 | 85 751 | 85 757 | 85 763 | 85 769 | 85 775 | 85 781 | 85 788 |
| 721 | 85 794 | 85 800 | 85 806 | 85 812 | 85 818 | 85 824 | 85 830 | 85 836 | 85 842 | 85 848 |
| 722 | 85 854 | 85 860 | 85 866 | 85 872 | 85 878 | 85 884 | 85 890 | 85 896 | 85 902 | 85 908 |
| 723 | 85 914 | 85 920 | 85 926 | 85 932 | 85 938 | 85 944 | 85 950 | 85 956 | 85 962 | 85 968 |
| 724 | 85 974 | 85 980 | 85 986 | 85 992 | 85 998 | 86 004 | 86 010 | 86 016 | 86 022 | 86 028 |
| 725 | 86 034 | 86 040 | 86 046 | 86 052 | 86 058 | 86 064 | 86 070 | 86 076 | 86 082 | 86 088 |
| 726 | 86 094 | 86 100 | 86 106 | 86 112 | 86 118 | 86 124 | 86 130 | 86 136 | 86 141 | 86 147 |
| 727 | 86 153 | 86 159 | 86 165 | 86 171 | 86 177 | 86 183 | 86 189 | 86 195 | 86 201 | 86 207 |
| 728 | 86 213 | 86 219 | 86 225 | 86 231 | 86 237 | 86 243 | 86 249 | 86 255 | 86 261 | 86 267 |
| 729 | 86 273 | 86 279 | 86 285 | 86 291 | 86 297 | 86 303 | 86 308 | 86 314 | 86 320 | 86 326 |

COMMON LOGARITHMS OF NUMBERS

(Continued)

| N | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
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| 730 | 86 332 | 86 338 | 86 344 | 86 350 | 86 356 | 86 362 | 86 368 | 86 374 | 86 380 | 86 386 |
| 731 | 86 392 | 86 398 | 86 404 | 86 410 | 86 415 | 86 421 | 86 427 | 86 433 | 86 439 | 86 445 |
| 732 | 86 451 | 86 457 | 86 463 | 86 469 | 86 475 | 86 481 | 86 487 | 86 493 | 86 499 | 86 504 |
| 733 | 86 510 | 86 516 | 86 522 | 86 528 | 86 534 | 86 540 | 86 546 | 86 552 | 86 558 | 86 564 |
| 734 | 86 570 | 86 576 | 86 581 | 86 587 | 86 593 | 86 599 | 86 605 | 86 611 | 86 617 | 86 623 |
| 735 | 86 629 | 86 635 | 86 641 | 86 646 | 86 652 | 86 658 | 86 664 | 86 670 | 86 676 | 86 682 |
| 736 | 86 688 | 86 694 | 86 700 | 86 705 | 86 711 | 86 717 | 86 723 | 86 729 | 86 735 | 86 741 |
| 737 | 86 747 | 86 753 | 86 759 | 86 764 | 86 770 | 86 776 | 86 782 | 86 788 | 86 794 | 86 800 |
| 738 | 86 806 | 86 812 | 86 817 | 86 823 | 86 829 | 86 835 | 86 841 | 86 847 | 86 853 | 86 859 |
| 739 | 86 864 | 86 870 | 86 876 | 86 882 | 86 888 | 86 894 | 86 900 | 86 906 | 86 911 | 86 917 |
| 740 | 86 923 | 86 929 | 86 935 | 86 941 | 86 947 | 86 953 | 86 958 | 86 964 | 86 970 | 86 976 |
| 741 | 86 982 | 86 988 | 86 994 | 86 999 | 87 005 | 87 011 | 87 017 | 87 023 | 87 029 | 87 035 |
| 742 | 87 040 | 87 046 | 87 052 | 87 058 | 87 064 | 87 070 | 87 075 | 87 081 | 87 087 | 87 093 |
| 743 | 87 099 | 87 105 | 87 111 | 87 116 | 87 122 | 87 128 | 87 134 | 87 140 | 87 146 | 87 151 |
| 744 | 87 157 | 87 163 | 87 169 | 87 175 | 87 181 | 87 186 | 87 192 | 87 198 | 87 204 | 87 210 |
| 745 | 87 216 | 87 221 | 87 227 | 87 233 | 87 239 | 87 245 | 87 251 | 87 256 | 87 262 | 87 268 |
| 746 | 87 274 | 87 280 | 87 286 | 87 291 | 87 297 | 87 303 | 87 309 | 87 315 | 87 320 | 87 326 |
| 747 | 87 332 | 87 338 | 87 344 | 87 349 | 87 355 | 87 361 | 87 367 | 87 373 | 87 379 | 87 384 |
| 748 | 87 390 | 87 396 | 87 402 | 87 408 | 87 413 | 87 419 | 87 425 | 87 431 | 87 437 | 87 442 |
| 749 | 87 448 | 87 454 | 87 460 | 87 466 | 87 471 | 87 477 | 87 483 | 87 489 | 87 495 | 87 500 |
| 750 | 87 506 | 87 512 | 87 518 | 87 523 | 87 529 | 87 535 | 87 541 | 87 547 | 87 552 | 87 558 |
| 751 | 87 564 | 87 570 | 87 576 | 87 581 | 87 587 | 87 593 | 87 599 | 87 604 | 87 610 | 87 616 |
| 752 | 87 622 | 87 628 | 87 633 | 87 639 | 87 645 | 87 651 | 87 656 | 87 662 | 87 668 | 87 674 |
| 753 | 87 679 | 87 685 | 87 691 | 87 697 | 87 703 | 87 708 | 87 714 | 87 720 | 87 726 | 87 731 |
| 754 | 87 737 | 87 743 | 87 749 | 87 754 | 87 760 | 87 766 | 87 772 | 87 777 | 87 783 | 87 789 |
| 755 | 87 795 | 87 800 | 87 806 | 87 812 | 87 818 | 87 823 | 87 829 | 87 835 | 87 841 | 87 846 |
| 756 | 87 852 | 87 858 | 87 864 | 87 869 | 87 875 | 87 881 | 87 887 | 87 892 | 87 898 | 87 904 |
| 757 | 87 910 | 87 915 | 87 921 | 87 927 | 87 933 | 87 938 | 87 944 | 87 950 | 87 955 | 87 961 |
| 758 | 87 967 | 87 973 | 87 978 | 87 984 | 87 990 | 87 996 | 88 001 | 88 007 | 88 013 | 88 018 |
| 759 | 88 024 | 88 030 | 88 036 | 88 041 | 88 047 | 88 053 | 88 058 | 88 064 | 88 070 | 88 076 |
| 760 | 88 081 | 88 087 | 88 093 | 88 098 | 88 104 | 88 110 | 88 116 | 88 121 | 88 127 | 88 133 |
| 761 | 88 138 | 88 144 | 88 150 | 88 156 | 88 161 | 88 167 | 88 173 | 88 178 | 88 184 | 88 190 |
| 762 | 88 195 | 88 201 | 88 207 | 88 213 | 88 218 | 88 224 | 88 230 | 88 235 | 88 241 | 88 247 |
| 763 | 88 252 | 88 258 | 88 264 | 88 270 | 88 275 | 88 281 | 88 287 | 88 292 | 88 298 | 88 304 |
| 764 | 88 309 | 88 315 | 88 321 | 88 326 | 88 332 | 88 338 | 88 343 | 88 349 | 88 355 | 88 360 |
| 765 | 88 366 | 88 372 | 88 377 | 88 383 | 88 389 | 88 395 | 88 400 | 88 406 | 88 412 | 88 417 |
| 766 | 88 423 | 88 429 | 88 434 | 88 440 | 88 446 | 88 451 | 88 457 | 88 463 | 88 468 | 88 474 |
| 767 | 88 480 | 88 485 | 88 491 | 88 497 | 88 502 | 88 508 | 88 513 | 88 519 | 88 525 | 88 530 |
| 768 | 88 536 | 88 542 | 88 547 | 88 553 | 88 559 | 88 564 | 88 570 | 88 576 | 88 581 | 88 587 |
| 769 | 88 593 | 88 598 | 88 604 | 88 610 | 88 615 | 88 621 | 88 627 | 88 632 | 88 638 | 88 643 |
| 770 | 88 649 | 88 655 | 88 660 | 88 666 | 88 672 | 88 677 | 88 683 | 88 689 | 88 694 | 88 700 |
| 771 | 88 705 | 88 711 | 88 717 | 88 722 | 88 728 | 88 734 | 88 739 | 88 745 | 88 750 | 88 756 |
| 772 | 88 762 | 88 767 | 88 773 | 88 779 | 88 784 | 88 790 | 88 795 | 88 801 | 88 807 | 88 812 |
| 773 | 88 818 | 88 824 | 88 829 | 88 835 | 88 840 | 88 846 | 88 852 | 88 857 | 88 863 | 88 868 |
| 774 | 88 874 | 88 880 | 88 885 | 88 891 | 88 897 | 88 902 | 88 908 | 88 913 | 88 919 | 88 925 |

COMMON LOGARITHMS OF NUMBERS

(Continued)

| N | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|-----|--------|--------|--------|--------|--------|--------|--------|--------|--------|--------|
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| 776 | 88 986 | 88 992 | 88 997 | 89 003 | 89 009 | 89 014 | 89 020 | 89 025 | 89 031 | 89 037 |
| 777 | 89 042 | 89 048 | 89 053 | 89 059 | 89 064 | 89 070 | 89 076 | 89 081 | 89 087 | 89 092 |
| 778 | 89 098 | 89 104 | 89 109 | 89 115 | 89 120 | 89 126 | 89 131 | 89 137 | 89 143 | 89 148 |
| 779 | 89 154 | 89 159 | 89 165 | 89 170 | 89 176 | 89 182 | 89 187 | 89 193 | 89 198 | 89 204 |
| 780 | 89 209 | 89 215 | 89 221 | 89 226 | 89 232 | 89 237 | 89 243 | 89 248 | 89 254 | 89 260 |
| 781 | 89 265 | 89 271 | 89 276 | 89 282 | 89 287 | 89 293 | 89 298 | 89 304 | 89 310 | 89 315 |
| 782 | 89 321 | 89 326 | 89 332 | 89 337 | 89 343 | 89 348 | 89 354 | 89 360 | 89 365 | 89 371 |
| 783 | 89 376 | 89 382 | 89 387 | 89 393 | 89 398 | 89 404 | 89 409 | 89 415 | 89 421 | 89 426 |
| 784 | 89 432 | 89 437 | 89 443 | 89 448 | 89 454 | 89 459 | 89 465 | 89 470 | 89 476 | 89 481 |
| 785 | 89 487 | 89 492 | 89 498 | 89 504 | 89 509 | 89 515 | 89 520 | 89 526 | 89 531 | 89 537 |
| 786 | 89 542 | 89 548 | 89 553 | 89 559 | 89 564 | 89 570 | 89 575 | 89 581 | 89 586 | 89 592 |
| 787 | 89 597 | 89 603 | 89 609 | 89 614 | 89 620 | 89 625 | 89 631 | 89 636 | 89 642 | 89 647 |
| 788 | 89 653 | 89 658 | 89 664 | 89 669 | 89 675 | 89 680 | 89 686 | 89 691 | 89 697 | 89 702 |
| 789 | 89 708 | 89 713 | 89 719 | 89 724 | 89 730 | 89 735 | 89 741 | 89 746 | 89 752 | 89 757 |
| 790 | 89 763 | 89 768 | 89 774 | 89 779 | 89 785 | 89 790 | 89 796 | 89 801 | 89 807 | 89 812 |
| 791 | 89 818 | 89 823 | 89 829 | 89 834 | 89 840 | 89 845 | 89 851 | 89 856 | 89 862 | 89 867 |
| 792 | 89 873 | 89 878 | 89 883 | 89 889 | 89 894 | 89 900 | 89 905 | 89 911 | 89 916 | 89 922 |
| 793 | 89 927 | 89 933 | 89 938 | 89 944 | 89 949 | 89 955 | 89 960 | 89 966 | 89 971 | 89 977 |
| 794 | 89 982 | 89 988 | 89 993 | 89 998 | 90 004 | 90 009 | 90 015 | 90 020 | 90 026 | 90 031 |
| 795 | 90 037 | 90 042 | 90 048 | 90 053 | 90 059 | 90 064 | 90 069 | 90 075 | 90 080 | 90 086 |
| 796 | 90 091 | 90 097 | 90 102 | 90 108 | 90 113 | 90 119 | 90 124 | 90 129 | 90 135 | 90 140 |
| 797 | 90 146 | 90 151 | 90 157 | 90 162 | 90 168 | 90 173 | 90 179 | 90 184 | 90 189 | 90 195 |
| 798 | 90 200 | 90 206 | 90 211 | 90 217 | 90 222 | 90 227 | 90 233 | 90 238 | 90 244 | 90 249 |
| 799 | 90 255 | 90 260 | 90 266 | 90 271 | 90 276 | 90 282 | 90 287 | 90 293 | 90 298 | 90 304 |
| 800 | 90 309 | 90 314 | 90 320 | 90 325 | 90 331 | 90 336 | 90 342 | 90 347 | 90 352 | 90 358 |
| 801 | 90 363 | 90 369 | 90 374 | 90 380 | 90 385 | 90 390 | 90 396 | 90 401 | 90 407 | 90 412 |
| 802 | 90 417 | 90 423 | 90 428 | 90 434 | 90 439 | 90 445 | 90 450 | 90 455 | 90 461 | 90 466 |
| 803 | 90 472 | 90 477 | 90 482 | 90 488 | 90 493 | 90 499 | 90 504 | 90 509 | 90 515 | 90 520 |
| 804 | 90 526 | 90 531 | 90 536 | 90 542 | 90 547 | 90 553 | 90 558 | 90 563 | 90 569 | 90 574 |
| 805 | 90 580 | 90 585 | 90 590 | 90 596 | 90 601 | 90 607 | 90 612 | 90 617 | 90 623 | 90 628 |
| 806 | 90 634 | 90 639 | 90 644 | 90 650 | 90 655 | 90 660 | 90 666 | 90 671 | 90 677 | 90 682 |
| 807 | 90 687 | 90 693 | 90 698 | 90 703 | 90 709 | 90 714 | 90 720 | 90 725 | 90 730 | 90 736 |
| 808 | 90 741 | 90 747 | 90 752 | 90 757 | 90 763 | 90 768 | 90 773 | 90 779 | 90 784 | 90 789 |
| 809 | 90 795 | 90 800 | 90 806 | 90 811 | 90 816 | 90 822 | 90 827 | 90 832 | 90 838 | 90 843 |
| 810 | 90 849 | 90 854 | 90 859 | 90 865 | 90 870 | 90 875 | 90 881 | 90 886 | 90 891 | 90 897 |
| 811 | 90 902 | 90 907 | 90 913 | 90 918 | 90 924 | 90 929 | 90 934 | 90 940 | 90 945 | 90 950 |
| 812 | 90 956 | 90 961 | 90 966 | 90 972 | 90 977 | 90 982 | 90 988 | 90 993 | 90 998 | 91 004 |
| 813 | 91 009 | 91 014 | 91 020 | 91 025 | 91 030 | 91 036 | 91 041 | 91 046 | 91 052 | 91 057 |
| 814 | 91 062 | 91 068 | 91 073 | 91 078 | 91 084 | 91 089 | 91 094 | 91 100 | 91 105 | 91 110 |
| 815 | 91 116 | 91 121 | 91 126 | 91 132 | 91 137 | 91 142 | 91 148 | 91 153 | 91 158 | 91 164 |
| 816 | 91 169 | 91 174 | 91 180 | 91 185 | 91 190 | 91 196 | 91 201 | 91 206 | 91 212 | 91 217 |
| 817 | 91 222 | 91 228 | 91 233 | 91 238 | 91 243 | 91 249 | 91 254 | 91 259 | 91 265 | 91 270 |
| 818 | 91 275 | 91 281 | 91 286 | 91 291 | 91 297 | 91 302 | 91 307 | 91 312 | 91 318 | 91 323 |
| 819 | 91 328 | 91 334 | 91 339 | 91 344 | 91 350 | 91 355 | 91 360 | 91 365 | 91 371 | 91 376 |

COMMON LOGARITHMS OF NUMBERS

(Continued)

| N | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|-----|--------|--------|--------|--------|--------|--------|--------|--------|--------|--------|
| 820 | 91 381 | 91 387 | 91 392 | 91 397 | 91 403 | 91 408 | 91 413 | 91 418 | 91 424 | 91 429 |
| 821 | 91 434 | 91 440 | 91 445 | 91 450 | 91 455 | 91 461 | 91 466 | 91 471 | 91 477 | 91 482 |
| 822 | 91 487 | 91 492 | 91 498 | 91 503 | 91 508 | 91 514 | 91 519 | 91 524 | 91 529 | 91 535 |
| 823 | 91 540 | 91 545 | 91 551 | 91 556 | 91 561 | 91 566 | 91 572 | 91 577 | 91 582 | 91 587 |
| 824 | 91 593 | 91 598 | 91 603 | 91 609 | 91 614 | 91 619 | 91 624 | 91 630 | 91 635 | 91 640 |
| 825 | 91 645 | 91 651 | 91 656 | 91 661 | 91 666 | 91 672 | 91 677 | 91 682 | 91 687 | 91 693 |
| 826 | 91 698 | 91 703 | 91 709 | 91 714 | 91 719 | 91 724 | 91 730 | 91 735 | 91 740 | 91 745 |
| 827 | 91 751 | 91 756 | 91 761 | 91 766 | 91 772 | 91 777 | 91 782 | 91 787 | 91 793 | 91 798 |
| 828 | 91 803 | 91 808 | 91 814 | 91 819 | 91 824 | 91 829 | 91 834 | 91 840 | 91 845 | 91 850 |
| 829 | 91 855 | 91 861 | 91 866 | 91 871 | 91 876 | 91 882 | 91 887 | 91 892 | 91 897 | 91 903 |
| 830 | 91 908 | 91 913 | 91 918 | 91 924 | 91 929 | 91 934 | 91 939 | 91 944 | 91 950 | 91 955 |
| 831 | 91 960 | 91 965 | 91 971 | 91 976 | 91 981 | 91 986 | 91 991 | 91 997 | 92 002 | 92 007 |
| 832 | 92 012 | 92 018 | 92 023 | 92 028 | 92 033 | 92 038 | 92 044 | 92 049 | 92 054 | 92 059 |
| 833 | 92 065 | 92 070 | 92 075 | 92 080 | 92 085 | 92 091 | 92 096 | 92 101 | 92 106 | 92 111 |
| 834 | 92 117 | 92 122 | 92 127 | 92 132 | 92 137 | 92 143 | 92 148 | 92 153 | 92 158 | 92 163 |
| 835 | 92 169 | 92 174 | 92 179 | 92 184 | 92 189 | 92 195 | 92 200 | 92 205 | 92 210 | 92 215 |
| 836 | 92 221 | 92 226 | 92 231 | 92 236 | 92 241 | 92 247 | 92 252 | 92 257 | 92 262 | 92 267 |
| 837 | 92 273 | 92 278 | 92 283 | 92 288 | 92 293 | 92 298 | 92 304 | 92 309 | 92 314 | 92 319 |
| 838 | 92 324 | 92 330 | 92 335 | 92 340 | 92 345 | 92 350 | 92 355 | 92 361 | 92 366 | 92 371 |
| 839 | 92 376 | 92 381 | 92 387 | 92 392 | 92 397 | 92 402 | 92 407 | 92 412 | 92 418 | 92 423 |
| 840 | 92 428 | 92 433 | 92 438 | 92 443 | 92 449 | 92 454 | 92 459 | 92 464 | 92 469 | 92 474 |
| 841 | 92 480 | 92 485 | 92 490 | 92 495 | 92 500 | 92 505 | 92 511 | 92 516 | 92 521 | 92 526 |
| 842 | 92 531 | 92 536 | 92 542 | 92 547 | 92 552 | 92 557 | 92 562 | 92 567 | 92 572 | 92 578 |
| 843 | 92 583 | 92 588 | 92 593 | 92 598 | 92 603 | 92 609 | 92 614 | 92 619 | 92 624 | 92 629 |
| 844 | 92 634 | 92 639 | 92 645 | 92 650 | 92 655 | 92 660 | 92 665 | 92 670 | 92 675 | 92 681 |
| 845 | 92 686 | 92 691 | 92 696 | 92 701 | 92 706 | 92 711 | 92 716 | 92 722 | 92 727 | 92 732 |
| 846 | 92 737 | 92 742 | 92 747 | 92 752 | 92 758 | 92 763 | 92 768 | 92 773 | 92 778 | 92 783 |
| 847 | 92 788 | 92 793 | 92 799 | 92 804 | 92 809 | 92 814 | 92 819 | 92 824 | 92 829 | 92 834 |
| 848 | 92 840 | 92 845 | 92 850 | 92 855 | 92 860 | 92 865 | 92 870 | 92 875 | 92 881 | 92 886 |
| 849 | 92 891 | 92 896 | 92 901 | 92 906 | 92 911 | 92 916 | 92 921 | 92 927 | 92 932 | 92 937 |
| 850 | 92 942 | 92 947 | 92 952 | 92 957 | 92 962 | 92 967 | 92 973 | 92 978 | 92 983 | 92 988 |
| 851 | 92 993 | 92 998 | 93 003 | 93 008 | 93 013 | 93 018 | 93 024 | 93 029 | 93 034 | 93 039 |
| 852 | 93 044 | 93 049 | 93 054 | 93 059 | 93 064 | 93 069 | 93 075 | 93 080 | 93 085 | 93 090 |
| 853 | 93 095 | 93 100 | 93 105 | 93 110 | 93 115 | 93 120 | 93 125 | 93 131 | 93 136 | 93 141 |
| 854 | 93 146 | 93 151 | 93 156 | 93 161 | 93 166 | 93 171 | 93 176 | 93 181 | 93 186 | 93 192 |
| 855 | 93 197 | 93 202 | 93 207 | 93 212 | 93 217 | 93 222 | 93 227 | 93 232 | 93 237 | 93 242 |
| 856 | 93 247 | 93 252 | 93 258 | 93 263 | 93 268 | 93 273 | 93 278 | 93 283 | 93 288 | 93 293 |
| 857 | 93 298 | 93 303 | 93 308 | 93 313 | 93 318 | 93 323 | 93 328 | 93 334 | 93 339 | 93 344 |
| 858 | 93 349 | 93 354 | 93 359 | 93 364 | 93 369 | 93 374 | 93 379 | 93 384 | 93 389 | 93 394 |
| 859 | 93 399 | 93 404 | 93 409 | 93 414 | 93 420 | 93 425 | 93 430 | 93 435 | 93 440 | 93 445 |
| 860 | 93 450 | 93 455 | 93 460 | 93 465 | 93 470 | 93 475 | 93 480 | 93 485 | 93 490 | 93 495 |
| 861 | 93 500 | 93 505 | 93 510 | 93 515 | 93 520 | 93 526 | 93 531 | 93 536 | 93 541 | 93 546 |
| 862 | 93 551 | 93 556 | 93 561 | 93 566 | 93 571 | 93 576 | 93 581 | 93 586 | 93 591 | 93 596 |
| 863 | 93 601 | 93 606 | 93 611 | 93 616 | 93 621 | 93 626 | 93 631 | 93 636 | 93 641 | 93 646 |
| 864 | 93 651 | 93 656 | 93 661 | 93 666 | 93 671 | 93 676 | 93 682 | 93 687 | 93 692 | 93 697 |

COMMON LOGARITHMS OF NUMBERS

(Continued)

| <i>N</i> | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|----------|--------|--------|--------|--------|--------|--------|--------|--------|--------|--------|
| 865 | 93 702 | 93 707 | 93 712 | 93 717 | 93 722 | 93 727 | 93 732 | 93 737 | 93 742 | 93 747 |
| 866 | 93 752 | 93 757 | 93 762 | 93 767 | 93 772 | 93 777 | 93 782 | 93 787 | 93 792 | 93 797 |
| 867 | 93 802 | 93 807 | 93 812 | 93 817 | 93 822 | 93 827 | 93 832 | 93 837 | 93 842 | 93 847 |
| 868 | 93 852 | 93 857 | 93 862 | 93 867 | 93 872 | 93 877 | 93 882 | 93 887 | 93 892 | 93 897 |
| 869 | 93 902 | 93 907 | 93 912 | 93 917 | 93 922 | 93 927 | 93 932 | 93 937 | 93 942 | 93 947 |
| 870 | 93 952 | 93 957 | 93 962 | 93 967 | 93 972 | 93 977 | 93 982 | 93 987 | 93 992 | 93 997 |
| 871 | 94 002 | 94 007 | 94 012 | 94 017 | 94 022 | 94 027 | 94 032 | 94 037 | 94 042 | 94 047 |
| 872 | 94 052 | 94 057 | 94 062 | 94 067 | 94 072 | 94 077 | 94 082 | 94 086 | 94 091 | 94 096 |
| 873 | 94 101 | 94 106 | 94 111 | 94 116 | 94 121 | 94 126 | 94 131 | 94 136 | 94 141 | 94 146 |
| 874 | 94 151 | 94 156 | 94 161 | 94 166 | 94 171 | 94 176 | 94 181 | 94 186 | 94 191 | 94 196 |
| 875 | 94 201 | 94 206 | 94 211 | 94 216 | 94 221 | 94 226 | 94 231 | 94 236 | 94 240 | 94 245 |
| 876 | 94 250 | 94 255 | 94 260 | 94 265 | 94 270 | 94 275 | 94 280 | 94 285 | 94 290 | 94 295 |
| 877 | 94 300 | 94 305 | 94 310 | 94 315 | 94 320 | 94 325 | 94 330 | 94 335 | 94 340 | 94 345 |
| 878 | 94 349 | 94 354 | 94 359 | 94 364 | 94 369 | 94 374 | 94 379 | 94 384 | 94 389 | 94 394 |
| 879 | 94 399 | 94 404 | 94 409 | 94 414 | 94 419 | 94 424 | 94 429 | 94 433 | 94 438 | 94 443 |
| 880 | 94 448 | 94 453 | 94 458 | 94 463 | 94 468 | 94 473 | 94 478 | 94 483 | 94 488 | 94 493 |
| 881 | 94 498 | 94 503 | 94 507 | 94 512 | 94 517 | 94 522 | 94 527 | 94 532 | 94 537 | 94 542 |
| 882 | 94 547 | 94 552 | 94 557 | 94 562 | 94 567 | 94 571 | 94 576 | 94 581 | 94 586 | 94 591 |
| 883 | 94 596 | 94 601 | 94 606 | 94 611 | 94 616 | 94 621 | 94 626 | 94 630 | 94 635 | 94 640 |
| 884 | 94 645 | 94 650 | 94 655 | 94 660 | 94 665 | 94 670 | 94 675 | 94 680 | 94 685 | 94 689 |
| 885 | 94 694 | 94 699 | 94 704 | 94 709 | 94 714 | 94 719 | 94 724 | 94 729 | 94 734 | 94 738 |
| 886 | 94 743 | 94 748 | 94 753 | 94 758 | 94 763 | 94 768 | 94 773 | 94 778 | 94 783 | 94 787 |
| 887 | 94 792 | 94 797 | 94 802 | 94 807 | 94 812 | 94 817 | 94 822 | 94 827 | 94 832 | 94 836 |
| 888 | 94 841 | 94 846 | 94 851 | 94 856 | 94 861 | 94 866 | 94 871 | 94 876 | 94 880 | 94 885 |
| 889 | 94 890 | 94 895 | 94 900 | 94 905 | 94 910 | 94 915 | 94 919 | 94 924 | 94 929 | 94 934 |
| 890 | 94 939 | 94 944 | 94 949 | 94 954 | 94 959 | 94 963 | 94 968 | 94 973 | 94 978 | 94 983 |
| 891 | 94 988 | 94 993 | 94 998 | 95 002 | 95 007 | 95 012 | 95 017 | 95 022 | 95 027 | 95 032 |
| 892 | 95 036 | 95 041 | 95 046 | 95 051 | 95 056 | 95 061 | 95 066 | 95 071 | 95 075 | 95 080 |
| 893 | 95 085 | 95 090 | 95 095 | 95 100 | 95 105 | 95 109 | 95 114 | 95 119 | 95 124 | 95 129 |
| 894 | 95 134 | 95 139 | 95 143 | 95 148 | 95 153 | 95 158 | 95 163 | 95 168 | 95 173 | 95 177 |
| 895 | 95 182 | 95 187 | 95 192 | 95 197 | 95 202 | 95 207 | 95 211 | 95 216 | 95 221 | 95 226 |
| 896 | 95 231 | 95 236 | 95 240 | 95 245 | 95 250 | 95 255 | 95 260 | 95 265 | 95 270 | 95 274 |
| 897 | 95 279 | 95 284 | 95 289 | 95 294 | 95 299 | 95 303 | 95 308 | 95 313 | 95 318 | 95 323 |
| 898 | 95 328 | 95 332 | 95 337 | 95 342 | 95 347 | 95 352 | 95 357 | 95 361 | 95 366 | 95 371 |
| 899 | 95 376 | 95 381 | 95 386 | 95 390 | 95 395 | 95 400 | 95 405 | 95 410 | 95 415 | 95 419 |
| 900 | 95 424 | 95 429 | 95 434 | 95 439 | 95 444 | 95 448 | 95 453 | 95 458 | 95 463 | 95 468 |
| 901 | 95 472 | 95 477 | 95 482 | 95 487 | 95 492 | 95 497 | 95 501 | 95 506 | 95 511 | 95 516 |
| 902 | 95 521 | 95 525 | 95 530 | 95 535 | 95 540 | 95 545 | 95 550 | 95 554 | 95 559 | 95 564 |
| 903 | 95 569 | 95 574 | 95 578 | 95 583 | 95 588 | 95 593 | 95 598 | 95 602 | 95 607 | 95 612 |
| 904 | 95 617 | 95 622 | 95 626 | 95 631 | 95 636 | 95 641 | 95 646 | 95 650 | 95 655 | 95 660 |
| 905 | 95 665 | 95 670 | 95 674 | 95 679 | 95 684 | 95 689 | 95 694 | 95 698 | 95 703 | 95 708 |
| 906 | 95 713 | 95 718 | 95 722 | 95 727 | 95 732 | 95 737 | 95 742 | 95 746 | 95 751 | 95 756 |
| 907 | 95 761 | 95 766 | 95 770 | 95 775 | 95 780 | 95 785 | 95 789 | 95 794 | 95 799 | 95 804 |
| 908 | 95 809 | 95 813 | 95 818 | 95 823 | 95 828 | 95 832 | 95 837 | 95 842 | 95 847 | 95 852 |
| 909 | 95 856 | 95 861 | 95 866 | 95 871 | 95 875 | 95 880 | 95 885 | 95 890 | 95 895 | 95 899 |

COMMON LOGARITHMS OF NUMBERS

(Continued)

| N | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|-----|--------|--------|--------|--------|--------|--------|--------|--------|--------|--------|
| 910 | 95 904 | 95 909 | 95 914 | 95 918 | 95 923 | 95 928 | 95 933 | 95 938 | 95 942 | 95 947 |
| 911 | 95 952 | 95 957 | 95 961 | 95 966 | 95 971 | 95 976 | 95 980 | 95 985 | 95 990 | 95 995 |
| 912 | 95 999 | 96 004 | 96 009 | 96 014 | 96 019 | 96 023 | 96 028 | 96 033 | 96 038 | 96 042 |
| 913 | 96 047 | 96 052 | 96 057 | 96 061 | 96 066 | 96 071 | 96 076 | 96 080 | 96 085 | 96 090 |
| 914 | 96 095 | 96 099 | 96 104 | 96 109 | 96 114 | 96 118 | 96 123 | 96 128 | 96 133 | 96 137 |
| 915 | 96 142 | 96 147 | 96 152 | 96 156 | 96 161 | 96 166 | 96 171 | 96 175 | 96 180 | 96 185 |
| 916 | 96 190 | 96 194 | 96 199 | 96 204 | 96 209 | 96 213 | 96 218 | 96 223 | 96 227 | 96 232 |
| 917 | 96 237 | 96 242 | 96 246 | 96 251 | 96 256 | 96 261 | 96 265 | 96 270 | 96 275 | 96 280 |
| 918 | 96 284 | 96 289 | 96 294 | 96 298 | 96 303 | 96 308 | 96 313 | 96 317 | 96 322 | 96 327 |
| 919 | 96 332 | 96 336 | 96 341 | 96 346 | 96 350 | 96 355 | 96 360 | 96 365 | 96 369 | 96 374 |
| 920 | 96 379 | 96 384 | 96 388 | 96 393 | 96 398 | 96 402 | 96 407 | 96 412 | 96 417 | 96 421 |
| 921 | 96 426 | 96 431 | 96 435 | 96 440 | 96 445 | 96 450 | 96 454 | 96 459 | 96 464 | 96 468 |
| 922 | 96 473 | 96 478 | 96 483 | 96 487 | 96 492 | 96 497 | 96 501 | 96 506 | 96 511 | 96 515 |
| 923 | 96 520 | 96 525 | 96 530 | 96 534 | 96 539 | 96 544 | 96 548 | 96 553 | 96 558 | 96 562 |
| 924 | 96 567 | 96 572 | 96 577 | 96 581 | 96 586 | 96 591 | 96 595 | 96 600 | 96 605 | 96 609 |
| 925 | 96 614 | 96 619 | 96 624 | 96 628 | 96 633 | 96 638 | 96 642 | 96 647 | 96 652 | 96 656 |
| 926 | 96 661 | 96 666 | 96 670 | 96 675 | 96 680 | 96 685 | 96 689 | 96 694 | 96 699 | 96 703 |
| 927 | 96 708 | 96 713 | 96 717 | 96 722 | 96 727 | 96 731 | 96 736 | 96 741 | 96 745 | 96 750 |
| 928 | 96 755 | 96 759 | 96 764 | 96 769 | 96 774 | 96 778 | 96 783 | 96 788 | 96 792 | 96 797 |
| 929 | 96 802 | 96 806 | 96 811 | 96 816 | 96 820 | 96 825 | 96 830 | 96 834 | 96 839 | 96 844 |
| 930 | 96 848 | 96 853 | 96 858 | 96 862 | 96 867 | 96 872 | 96 876 | 96 881 | 96 886 | 96 890 |
| 931 | 96 895 | 96 900 | 96 904 | 96 909 | 96 914 | 96 918 | 96 923 | 96 928 | 96 932 | 96 937 |
| 932 | 96 942 | 96 946 | 96 951 | 96 956 | 96 960 | 96 965 | 96 970 | 96 974 | 96 979 | 96 984 |
| 933 | 96 988 | 96 993 | 96 997 | 97 002 | 97 007 | 97 011 | 97 016 | 97 021 | 97 025 | 97 030 |
| 934 | 97 035 | 97 039 | 97 044 | 97 049 | 97 053 | 97 058 | 97 063 | 97 067 | 97 072 | 97 077 |
| 935 | 97 081 | 97 086 | 97 090 | 97 095 | 97 100 | 97 104 | 97 109 | 97 114 | 97 118 | 97 123 |
| 936 | 97 128 | 97 132 | 97 137 | 97 142 | 97 146 | 97 151 | 97 155 | 97 160 | 97 165 | 97 169 |
| 937 | 97 174 | 97 179 | 97 183 | 97 188 | 97 192 | 97 197 | 97 202 | 97 206 | 97 211 | 97 216 |
| 938 | 97 220 | 97 225 | 97 230 | 97 234 | 97 239 | 97 243 | 97 248 | 97 253 | 97 257 | 97 262 |
| 939 | 97 267 | 97 271 | 97 276 | 97 280 | 97 285 | 97 290 | 97 294 | 97 299 | 97 304 | 97 308 |
| 940 | 97 313 | 97 317 | 97 322 | 97 327 | 97 331 | 97 336 | 97 340 | 97 345 | 97 350 | 97 354 |
| 941 | 97 359 | 97 364 | 97 368 | 97 373 | 97 377 | 97 382 | 97 387 | 97 391 | 97 396 | 97 400 |
| 942 | 97 405 | 97 410 | 97 414 | 97 419 | 97 424 | 97 428 | 97 433 | 97 437 | 97 442 | 97 447 |
| 943 | 97 451 | 97 456 | 97 460 | 97 465 | 97 470 | 97 474 | 97 479 | 97 483 | 97 488 | 97 493 |
| 944 | 97 497 | 97 502 | 97 506 | 97 511 | 97 516 | 97 520 | 97 525 | 97 529 | 97 534 | 97 539 |
| 945 | 97 543 | 97 548 | 97 552 | 97 557 | 97 562 | 97 566 | 97 571 | 97 575 | 97 580 | 97 585 |
| 946 | 97 589 | 97 594 | 97 598 | 97 603 | 97 607 | 97 612 | 97 617 | 97 621 | 97 626 | 97 630 |
| 947 | 97 635 | 97 640 | 97 644 | 97 649 | 97 653 | 97 658 | 97 663 | 97 667 | 97 672 | 97 676 |
| 948 | 97 681 | 97 685 | 97 690 | 97 695 | 97 699 | 97 704 | 97 708 | 97 713 | 97 717 | 97 722 |
| 949 | 97 727 | 97 731 | 97 736 | 97 740 | 97 745 | 97 749 | 97 754 | 97 759 | 97 763 | 97 768 |
| 950 | 97 772 | 97 777 | 97 782 | 97 786 | 97 791 | 97 795 | 97 800 | 97 804 | 97 809 | 97 813 |
| 951 | 97 818 | 97 823 | 97 827 | 97 832 | 97 836 | 97 841 | 97 845 | 97 850 | 97 855 | 97 859 |
| 952 | 97 864 | 97 868 | 97 873 | 97 877 | 97 882 | 97 886 | 97 891 | 97 896 | 97 900 | 97 905 |
| 953 | 97 909 | 97 914 | 97 918 | 97 923 | 97 928 | 97 932 | 97 937 | 97 941 | 97 946 | 97 950 |
| 954 | 97 955 | 97 959 | 97 964 | 97 968 | 97 973 | 97 978 | 97 982 | 97 987 | 97 991 | 97 996 |

COMMON LOGARITHMS OF NUMBERS

(Continued)

| N | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|------|--------|--------|--------|--------|--------|--------|--------|--------|--------|--------|
| 955 | 98 000 | 98 005 | 98 009 | 98 014 | 98 019 | 98 023 | 98 028 | 98 032 | 98 037 | 98 041 |
| 956 | 98 046 | 98 050 | 98 055 | 98 059 | 98 064 | 98 068 | 98 073 | 98 078 | 98 082 | 98 087 |
| 957 | 98 091 | 98 096 | 98 100 | 98 105 | 98 109 | 98 114 | 98 118 | 98 123 | 98 127 | 98 132 |
| 958 | 98 137 | 98 141 | 98 146 | 98 150 | 98 155 | 98 159 | 98 164 | 98 168 | 98 173 | 98 177 |
| 959 | 98 182 | 98 186 | 98 191 | 98 195 | 98 200 | 98 204 | 98 209 | 98 214 | 98 218 | 98 223 |
| 960 | 98 227 | 98 232 | 98 236 | 98 241 | 98 245 | 98 250 | 98 254 | 98 259 | 98 263 | 98 268 |
| 961 | 98 272 | 98 277 | 98 281 | 98 286 | 98 290 | 98 295 | 98 299 | 98 304 | 98 308 | 98 313 |
| 962 | 98 318 | 98 322 | 98 327 | 98 331 | 98 336 | 98 340 | 98 345 | 98 349 | 98 354 | 98 358 |
| 963 | 98 363 | 98 367 | 98 372 | 98 376 | 98 381 | 98 385 | 98 390 | 98 394 | 98 399 | 98 403 |
| 964 | 98 408 | 98 412 | 98 417 | 98 421 | 98 426 | 98 430 | 98 435 | 98 439 | 98 444 | 98 448 |
| 965 | 98 453 | 98 457 | 98 462 | 98 466 | 98 471 | 98 475 | 98 480 | 98 484 | 98 489 | 98 493 |
| 966 | 98 498 | 98 502 | 98 507 | 98 511 | 98 516 | 98 520 | 98 525 | 98 529 | 98 534 | 98 538 |
| 967 | 98 543 | 98 547 | 98 552 | 98 556 | 98 561 | 98 565 | 98 570 | 98 574 | 98 579 | 98 583 |
| 968 | 98 588 | 98 592 | 98 597 | 98 601 | 98 605 | 98 610 | 98 614 | 98 619 | 98 623 | 98 628 |
| 969 | 98 632 | 98 637 | 98 641 | 98 646 | 98 650 | 98 655 | 98 659 | 98 664 | 98 668 | 98 673 |
| 970 | 98 677 | 98 682 | 98 686 | 98 691 | 98 695 | 98 700 | 98 704 | 98 709 | 98 713 | 98 717 |
| 971 | 98 722 | 98 726 | 98 731 | 98 735 | 98 740 | 98 744 | 98 749 | 98 753 | 98 758 | 98 762 |
| 972 | 98 767 | 98 771 | 98 776 | 98 780 | 98 784 | 98 789 | 98 793 | 98 798 | 98 802 | 98 807 |
| 973 | 98 811 | 98 816 | 98 820 | 98 825 | 98 829 | 98 834 | 98 838 | 98 843 | 98 847 | 98 851 |
| 974 | 98 856 | 98 860 | 98 865 | 98 869 | 98 874 | 98 878 | 98 883 | 98 887 | 98 892 | 98 896 |
| 975 | 98 900 | 98 905 | 98 909 | 98 914 | 98 918 | 98 923 | 98 927 | 98 932 | 98 936 | 98 941 |
| 976 | 98 945 | 98 949 | 98 954 | 98 958 | 98 963 | 98 967 | 98 972 | 98 976 | 98 981 | 98 985 |
| 977 | 98 989 | 98 994 | 98 998 | 99 003 | 99 007 | 99 012 | 99 016 | 99 021 | 99 025 | 99 029 |
| 978 | 99 034 | 99 038 | 99 043 | 99 047 | 99 052 | 99 056 | 99 061 | 99 065 | 99 069 | 99 074 |
| 979 | 99 078 | 99 083 | 99 087 | 99 092 | 99 096 | 99 100 | 99 105 | 99 109 | 99 114 | 99 118 |
| 980 | 99 123 | 99 127 | 99 131 | 99 136 | 99 140 | 99 145 | 99 149 | 99 154 | 99 158 | 99 162 |
| 981 | 99 167 | 99 171 | 99 176 | 99 180 | 99 185 | 99 189 | 99 193 | 99 198 | 99 202 | 99 207 |
| 982 | 99 211 | 99 216 | 99 220 | 99 224 | 99 229 | 99 233 | 99 238 | 99 242 | 99 247 | 99 251 |
| 983 | 99 255 | 99 260 | 99 264 | 99 269 | 99 273 | 99 277 | 99 282 | 99 286 | 99 291 | 99 295 |
| 984 | 99 300 | 99 304 | 99 308 | 99 313 | 99 317 | 99 322 | 99 326 | 99 330 | 99 335 | 99 339 |
| 985 | 99 344 | 99 348 | 99 352 | 99 357 | 99 361 | 99 366 | 99 370 | 99 374 | 99 379 | 99 383 |
| 986 | 99 388 | 99 392 | 99 396 | 99 401 | 99 405 | 99 410 | 99 414 | 99 419 | 99 423 | 99 427 |
| 987 | 99 432 | 99 436 | 99 441 | 99 445 | 99 449 | 99 454 | 99 458 | 99 463 | 99 467 | 99 471 |
| 988 | 99 476 | 99 480 | 99 484 | 99 489 | 99 493 | 99 498 | 99 502 | 99 506 | 99 511 | 99 515 |
| 989 | 99 520 | 99 524 | 99 528 | 99 533 | 99 537 | 99 542 | 99 546 | 99 550 | 99 555 | 99 559 |
| 990 | 99 564 | 99 568 | 99 572 | 99 577 | 99 581 | 99 585 | 99 590 | 99 594 | 99 599 | 99 603 |
| 991 | 99 607 | 99 612 | 99 616 | 99 621 | 99 625 | 99 629 | 99 634 | 99 638 | 99 642 | 99 647 |
| 992 | 99 651 | 99 656 | 99 660 | 99 664 | 99 669 | 99 673 | 99 677 | 99 682 | 99 686 | 99 691 |
| 993 | 99 695 | 99 699 | 99 704 | 99 708 | 99 712 | 99 717 | 99 721 | 99 726 | 99 730 | 99 734 |
| 994 | 99 739 | 99 743 | 99 747 | 99 752 | 99 756 | 99 760 | 99 765 | 99 769 | 99 774 | 99 778 |
| 995 | 99 782 | 99 787 | 99 791 | 99 795 | 99 800 | 99 804 | 99 808 | 99 813 | 99 817 | 99 822 |
| 996 | 99 826 | 99 830 | 99 835 | 99 839 | 99 843 | 99 848 | 99 852 | 99 856 | 99 861 | 99 865 |
| 997 | 99 870 | 99 874 | 99 878 | 99 883 | 99 887 | 99 891 | 99 896 | 99 900 | 99 904 | 99 909 |
| 998 | 99 913 | 99 917 | 99 922 | 99 926 | 99 930 | 99 935 | 99 939 | 99 944 | 99 948 | 99 952 |
| 999 | 99 957 | 99 961 | 99 965 | 99 970 | 99 974 | 99 978 | 99 983 | 99 987 | 99 991 | 99 996 |
| 1000 | 00 000 | 00 004 | 00 009 | 00 013 | 00 017 | 00 022 | 00 026 | 00 030 | 00 035 | 00 039 |

NATURAL LOGARITHMS OF NUMBERS FROM
1 TO 10 (Base e)

| N | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|-----|--------|--------|--------|--------|--------|--------|--------|--------|--------|--------|
| 1.0 | 0.0000 | 0.0099 | 0.0198 | 0.0296 | 0.0392 | 0.0488 | 0.0583 | 0.0677 | 0.0770 | 0.0862 |
| 1.1 | 0.0953 | 0.1044 | 0.1133 | 0.1222 | 0.1310 | 0.1398 | 0.1484 | 0.1570 | 0.1655 | 0.1740 |
| 1.2 | 0.1823 | 0.1906 | 0.1989 | 0.2070 | 0.2151 | 0.2231 | 0.2311 | 0.2390 | 0.2469 | 0.2546 |
| 1.3 | 0.2624 | 0.2700 | 0.2776 | 0.2852 | 0.2927 | 0.3001 | 0.3075 | 0.3148 | 0.3221 | 0.3293 |
| 1.4 | 0.3365 | 0.3436 | 0.3507 | 0.3577 | 0.3646 | 0.3716 | 0.3784 | 0.3853 | 0.3920 | 0.3988 |
| 1.5 | 0.4055 | 0.4121 | 0.4187 | 0.4253 | 0.4318 | 0.4383 | 0.4447 | 0.4511 | 0.4574 | 0.4637 |
| 1.6 | 0.4700 | 0.4762 | 0.4824 | 0.4886 | 0.4947 | 0.5008 | 0.5068 | 0.5128 | 0.5188 | 0.5247 |
| 1.7 | 0.5306 | 0.5365 | 0.5423 | 0.5481 | 0.5539 | 0.5596 | 0.5653 | 0.5710 | 0.5766 | 0.5822 |
| 1.8 | 0.5878 | 0.5933 | 0.5988 | 0.6043 | 0.6098 | 0.6152 | 0.6206 | 0.6258 | 0.6313 | 0.6366 |
| 1.9 | 0.6419 | 0.6471 | 0.6523 | 0.6575 | 0.6627 | 0.6678 | 0.6729 | 0.6780 | 0.6831 | 0.6881 |
| 2.0 | 0.6932 | 0.6981 | 0.7031 | 0.7080 | 0.7130 | 0.7178 | 0.7227 | 0.7276 | 0.7324 | 0.7372 |
| 2.1 | 0.7419 | 0.7467 | 0.7514 | 0.7561 | 0.7608 | 0.7655 | 0.7701 | 0.7747 | 0.7793 | 0.7839 |
| 2.2 | 0.7885 | 0.7930 | 0.7975 | 0.8020 | 0.8065 | 0.8109 | 0.8154 | 0.8198 | 0.8242 | 0.8286 |
| 2.3 | 0.8329 | 0.8373 | 0.8416 | 0.8459 | 0.8502 | 0.8544 | 0.8587 | 0.8629 | 0.8671 | 0.8713 |
| 2.4 | 0.8755 | 0.8796 | 0.8838 | 0.8879 | 0.8920 | 0.8961 | 0.9001 | 0.9042 | 0.9083 | 0.9123 |
| 2.5 | 0.9163 | 0.9203 | 0.9243 | 0.9282 | 0.9322 | 0.9361 | 0.9400 | 0.9439 | 0.9478 | 0.9517 |
| 2.6 | 0.9555 | 0.9594 | 0.9632 | 0.9670 | 0.9708 | 0.9746 | 0.9783 | 0.9820 | 0.9858 | 0.9895 |
| 2.7 | 0.9933 | 0.9970 | 1.0006 | 1.0043 | 1.0080 | 1.0116 | 1.0152 | 1.0189 | 1.0225 | 1.0260 |
| 2.8 | 1.0296 | 1.0332 | 1.0367 | 1.0403 | 1.0438 | 1.0473 | 1.0508 | 1.0543 | 1.0578 | 1.0613 |
| 2.9 | 1.0647 | 1.0681 | 1.0716 | 1.0750 | 1.0784 | 1.0818 | 1.0852 | 1.0886 | 1.0919 | 1.0953 |
| 3.0 | 1.0986 | 1.1019 | 1.1053 | 1.1086 | 1.1119 | 1.1151 | 1.1184 | 1.1217 | 1.1249 | 1.1282 |
| 3.1 | 1.1314 | 1.1346 | 1.1378 | 1.1410 | 1.1442 | 1.1474 | 1.1506 | 1.1537 | 1.1569 | 1.1600 |
| 3.2 | 1.1632 | 1.1663 | 1.1694 | 1.1725 | 1.1756 | 1.1787 | 1.1817 | 1.1848 | 1.1878 | 1.1909 |
| 3.3 | 1.1939 | 1.1970 | 1.2000 | 1.2030 | 1.2060 | 1.2090 | 1.2119 | 1.2149 | 1.2179 | 1.2208 |
| 3.4 | 1.2238 | 1.2267 | 1.2296 | 1.2326 | 1.2355 | 1.2384 | 1.2413 | 1.2442 | 1.2470 | 1.2499 |
| 3.5 | 1.2528 | 1.2556 | 1.2585 | 1.2613 | 1.2641 | 1.2670 | 1.2698 | 1.2726 | 1.2754 | 1.2782 |
| 3.6 | 1.2809 | 1.2837 | 1.2865 | 1.2892 | 1.2920 | 1.2947 | 1.2975 | 1.3002 | 1.3029 | 1.3056 |
| 3.7 | 1.3083 | 1.3110 | 1.3137 | 1.3164 | 1.3191 | 1.3218 | 1.3244 | 1.3271 | 1.3297 | 1.3324 |
| 3.8 | 1.3350 | 1.3376 | 1.3403 | 1.3429 | 1.3455 | 1.3481 | 1.3507 | 1.3533 | 1.3558 | 1.3584 |
| 3.9 | 1.3610 | 1.3635 | 1.3661 | 1.3686 | 1.3712 | 1.3737 | 1.3762 | 1.3788 | 1.3813 | 1.3838 |
| 4.0 | 1.3863 | 1.3888 | 1.3913 | 1.3938 | 1.3962 | 1.3987 | 1.4012 | 1.4036 | 1.4061 | 1.4085 |
| 4.1 | 1.4110 | 1.4134 | 1.4159 | 1.4183 | 1.4207 | 1.4231 | 1.4255 | 1.4279 | 1.4303 | 1.4327 |
| 4.2 | 1.4351 | 1.4375 | 1.4398 | 1.4422 | 1.4446 | 1.4469 | 1.4493 | 1.4516 | 1.4540 | 1.4563 |
| 4.3 | 1.4586 | 1.4609 | 1.4633 | 1.4656 | 1.4679 | 1.4702 | 1.4725 | 1.4748 | 1.4770 | 1.4793 |
| 4.4 | 1.4816 | 1.4839 | 1.4861 | 1.4884 | 1.4907 | 1.4929 | 1.4951 | 1.4974 | 1.4996 | 1.5019 |
| 4.5 | 1.5041 | 1.5063 | 1.5085 | 1.5107 | 1.5129 | 1.5151 | 1.5173 | 1.5195 | 1.5217 | 1.5239 |
| 4.6 | 1.5261 | 1.5282 | 1.5304 | 1.5326 | 1.5347 | 1.5369 | 1.5390 | 1.5412 | 1.5433 | 1.5454 |
| 4.7 | 1.5476 | 1.5497 | 1.5518 | 1.5539 | 1.5560 | 1.5581 | 1.5603 | 1.5624 | 1.5644 | 1.5665 |
| 4.8 | 1.5686 | 1.5707 | 1.5728 | 1.5749 | 1.5769 | 1.5790 | 1.5810 | 1.5831 | 1.5852 | 1.5872 |
| 4.9 | 1.5892 | 1.5913 | 1.5933 | 1.5953 | 1.5974 | 1.5994 | 1.6014 | 1.6034 | 1.6054 | 1.6074 |
| 5.0 | 1.6094 | 1.6114 | 1.6134 | 1.6154 | 1.6174 | 1.6194 | 1.6214 | 1.6233 | 1.6253 | 1.6273 |
| 5.1 | 1.6292 | 1.6312 | 1.6332 | 1.6351 | 1.6371 | 1.6390 | 1.6409 | 1.6429 | 1.6448 | 1.6467 |
| 5.2 | 1.6487 | 1.6506 | 1.6525 | 1.6545 | 1.6563 | 1.6582 | 1.6601 | 1.6620 | 1.6639 | 1.6658 |
| 5.3 | 1.6677 | 1.6696 | 1.6715 | 1.6734 | 1.6753 | 1.6771 | 1.6790 | 1.6808 | 1.6827 | 1.6846 |
| 5.4 | 1.6864 | 1.6883 | 1.6901 | 1.6919 | 1.6938 | 1.6956 | 1.6975 | 1.6993 | 1.7011 | 1.7029 |

NATURAL LOGARITHMS OF NUMBERS

(Continued)

| N | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|-----|--------|--------|--------|--------|--------|--------|--------|--------|--------|--------|
| 5.5 | 1.7048 | 1.7066 | 1.7084 | 1.7102 | 1.7120 | 1.7138 | 1.7156 | 1.7174 | 1.7192 | 1.7210 |
| 5.6 | 1.7228 | 1.7246 | 1.7263 | 1.7281 | 1.7299 | 1.7317 | 1.7334 | 1.7352 | 1.7370 | 1.7387 |
| 5.7 | 1.7405 | 1.7422 | 1.7440 | 1.7457 | 1.7475 | 1.7491 | 1.7509 | 1.7527 | 1.7544 | 1.7561 |
| 5.8 | 1.7579 | 1.7596 | 1.7613 | 1.7630 | 1.7647 | 1.7664 | 1.7682 | 1.7699 | 1.7716 | 1.7733 |
| 5.9 | 1.7750 | 1.7767 | 1.7783 | 1.7800 | 1.7817 | 1.7834 | 1.7851 | 1.7868 | 1.7884 | 1.7901 |
| 6.0 | 1.7918 | 1.7934 | 1.7951 | 1.7968 | 1.7984 | 1.8001 | 1.8017 | 1.8034 | 1.8050 | 1.8067 |
| 6.1 | 1.8083 | 1.8099 | 1.8116 | 1.8132 | 1.8148 | 1.8165 | 1.8181 | 1.8197 | 1.8213 | 1.8229 |
| 6.2 | 1.8246 | 1.8262 | 1.8278 | 1.8294 | 1.8310 | 1.8326 | 1.8342 | 1.8358 | 1.8374 | 1.8390 |
| 6.3 | 1.8406 | 1.8421 | 1.8437 | 1.8453 | 1.8469 | 1.8485 | 1.8500 | 1.8516 | 1.8532 | 1.8547 |
| 6.4 | 1.8563 | 1.8579 | 1.8594 | 1.8610 | 1.8625 | 1.8641 | 1.8656 | 1.8672 | 1.8687 | 1.8703 |
| 6.5 | 1.8718 | 1.8733 | 1.8749 | 1.8764 | 1.8779 | 1.8795 | 1.8810 | 1.8825 | 1.8840 | 1.8856 |
| 6.6 | 1.8871 | 1.8886 | 1.8901 | 1.8916 | 1.8931 | 1.8946 | 1.8961 | 1.8976 | 1.8991 | 1.9006 |
| 6.7 | 1.9021 | 1.9036 | 1.9051 | 1.9066 | 1.9081 | 1.9095 | 1.9110 | 1.9125 | 1.9140 | 1.9155 |
| 6.8 | 1.9169 | 1.9184 | 1.9199 | 1.9213 | 1.9228 | 1.9243 | 1.9257 | 1.9272 | 1.9286 | 1.9301 |
| 6.9 | 1.9315 | 1.9330 | 1.9344 | 1.9359 | 1.9373 | 1.9387 | 1.9402 | 1.9416 | 1.9431 | 1.9445 |
| 7.0 | 1.9459 | 1.9473 | 1.9488 | 1.9502 | 1.9516 | 1.9530 | 1.9545 | 1.9559 | 1.9573 | 1.9587 |
| 7.1 | 1.9601 | 1.9615 | 1.9629 | 1.9643 | 1.9657 | 1.9671 | 1.9685 | 1.9699 | 1.9713 | 1.9727 |
| 7.2 | 1.9741 | 1.9755 | 1.9769 | 1.9782 | 1.9796 | 1.9810 | 1.9824 | 1.9838 | 1.9851 | 1.9865 |
| 7.3 | 1.9879 | 1.9892 | 1.9906 | 1.9920 | 1.9933 | 1.9947 | 1.9961 | 1.9974 | 1.9988 | 2.0001 |
| 7.4 | 2.0015 | 2.0028 | 2.0042 | 2.0055 | 2.0069 | 2.0082 | 2.0096 | 2.0109 | 2.0122 | 2.0136 |
| 7.5 | 2.0149 | 2.0162 | 2.0176 | 2.0189 | 2.0202 | 2.0216 | 2.0229 | 2.0242 | 2.0255 | 2.0268 |
| 7.6 | 2.0282 | 2.0295 | 2.0308 | 2.0321 | 2.0334 | 2.0347 | 2.0360 | 2.0373 | 2.0386 | 2.0399 |
| 7.7 | 2.0412 | 2.0425 | 2.0438 | 2.0451 | 2.0464 | 2.0477 | 2.0490 | 2.0503 | 2.0516 | 2.0528 |
| 7.8 | 2.0541 | 2.0554 | 2.0567 | 2.0580 | 2.0592 | 2.0605 | 2.0618 | 2.0631 | 2.0643 | 2.0656 |
| 7.9 | 2.0669 | 2.0681 | 2.0694 | 2.0707 | 2.0719 | 2.0732 | 2.0744 | 2.0757 | 2.0769 | 2.0782 |
| 8.0 | 2.0794 | 2.0807 | 2.0819 | 2.0832 | 2.0844 | 2.0857 | 2.0869 | 2.0882 | 2.0894 | 2.0906 |
| 8.1 | 2.0919 | 2.0931 | 2.0943 | 2.0956 | 2.0968 | 2.0980 | 2.0992 | 2.1005 | 2.1017 | 2.1029 |
| 8.2 | 2.1041 | 2.1054 | 2.1066 | 2.1078 | 2.1090 | 2.1102 | 2.1114 | 2.1126 | 2.1138 | 2.1151 |
| 8.3 | 2.1163 | 2.1175 | 2.1187 | 2.1199 | 2.1211 | 2.1223 | 2.1235 | 2.1247 | 2.1259 | 2.1270 |
| 8.4 | 2.1282 | 2.1294 | 2.1306 | 2.1318 | 2.1330 | 2.1342 | 2.1354 | 2.1365 | 2.1377 | 2.1389 |
| 8.5 | 2.1401 | 2.1412 | 2.1424 | 2.1436 | 2.1448 | 2.1459 | 2.1471 | 2.1483 | 2.1494 | 2.1506 |
| 8.6 | 2.1518 | 2.1529 | 2.1541 | 2.1552 | 2.1564 | 2.1576 | 2.1587 | 2.1599 | 2.1610 | 2.1622 |
| 8.7 | 2.1633 | 2.1645 | 2.1656 | 2.1668 | 2.1679 | 2.1691 | 2.1702 | 2.1713 | 2.1725 | 2.1736 |
| 8.8 | 2.1748 | 2.1759 | 2.1770 | 2.1782 | 2.1793 | 2.1804 | 2.1816 | 2.1827 | 2.1838 | 2.1849 |
| 8.9 | 2.1861 | 2.1872 | 2.1883 | 2.1894 | 2.1905 | 2.1917 | 2.1928 | 2.1939 | 2.1950 | 2.1961 |
| 9.0 | 2.1972 | 2.1983 | 2.1994 | 2.2006 | 2.2017 | 2.2028 | 2.2039 | 2.2050 | 2.2061 | 2.2072 |
| 9.1 | 2.2083 | 2.2094 | 2.2105 | 2.2116 | 2.2127 | 2.2138 | 2.2149 | 2.2159 | 2.2170 | 2.2181 |
| 9.2 | 2.2192 | 2.2203 | 2.2214 | 2.2225 | 2.2235 | 2.2246 | 2.2257 | 2.2268 | 2.2279 | 2.2289 |
| 9.3 | 2.2300 | 2.2311 | 2.2322 | 2.2332 | 2.2343 | 2.2354 | 2.2365 | 2.2375 | 2.2386 | 2.2397 |
| 9.4 | 2.2407 | 2.2418 | 2.2428 | 2.2439 | 2.2450 | 2.2460 | 2.2471 | 2.2481 | 2.2492 | 2.2502 |
| 9.5 | 2.2513 | 2.2523 | 2.2534 | 2.2544 | 2.2555 | 2.2565 | 2.2576 | 2.2586 | 2.2597 | 2.2607 |
| 9.6 | 2.2618 | 2.2628 | 2.2638 | 2.2649 | 2.2659 | 2.2670 | 2.2680 | 2.2690 | 2.2701 | 2.2711 |
| 9.7 | 2.2721 | 2.2732 | 2.2742 | 2.2752 | 2.2762 | 2.2773 | 2.2783 | 2.2793 | 2.2803 | 2.2814 |
| 9.8 | 2.2824 | 2.2834 | 2.2844 | 2.2854 | 2.2865 | 2.2875 | 2.2885 | 2.2895 | 2.2905 | 2.2915 |
| 9.9 | 2.2925 | 2.2935 | 2.2946 | 2.2956 | 2.2966 | 2.2976 | 2.2986 | 2.2996 | 2.3006 | 2.3016 |

NATURAL LOGARITHMS (EACH INCREASED
BY 10) OF NUMBERS FROM 0.00 TO 0.99

| No. | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|-----|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|
| 0.0 | | 5.395 | 6.088 | 6.493 | 6.781 | 7.004 | 7.187 | 7.341 | 7.474 | 7.592 |
| 0.1 | 7.697 | 7.793 | 7.880 | 7.960 | 8.034 | 8.103 | 8.167 | 8.228 | 8.285 | 8.339 |
| 0.2 | 8.391 | 8.439 | 8.486 | 8.530 | 8.573 | 8.614 | 8.653 | 8.691 | 8.727 | 8.762 |
| 0.3 | 8.796 | 8.829 | 8.861 | 8.891 | 8.921 | 8.950 | 8.978 | 9.006 | 9.032 | 9.058 |
| 0.4 | 9.084 | 9.108 | 9.132 | 9.156 | 9.179 | 9.201 | 9.223 | 9.245 | 9.266 | 9.287 |
| 0.5 | 9.307 | 9.327 | 9.346 | 9.365 | 9.384 | 9.402 | 9.420 | 9.438 | 9.455 | 9.472 |
| 0.6 | 9.489 | 9.506 | 9.522 | 9.538 | 9.554 | 9.569 | 9.584 | 9.600 | 9.614 | 9.629 |
| 0.7 | 9.643 | 9.658 | 9.671 | 9.685 | 9.699 | 9.712 | 9.726 | 9.739 | 9.752 | 9.764 |
| 0.8 | 9.777 | 9.789 | 9.802 | 9.814 | 9.826 | 9.837 | 9.849 | 9.861 | 9.872 | 9.883 |
| 0.9 | 9.895 | 9.906 | 9.917 | 9.927 | 9.938 | 9.949 | 9.959 | 9.970 | 9.980 | 9.990 |

NATURAL LOGARITHMS OF WHOLE NUMBERS
FROM 10 TO 209

| No. | 0 | 1 | 2 | 3 | 4 | 5 | 6 | 7 | 8 | 9 |
|-----|-------|-------|-------|-------|-------|-------|-------|-------|-------|-------|
| 1 | 2.303 | 2.398 | 2.485 | 2.565 | 2.639 | 2.708 | 2.773 | 2.833 | 2.890 | 2.944 |
| 2 | 2.996 | 3.045 | 3.091 | 3.136 | 3.178 | 3.219 | 3.258 | 3.296 | 3.332 | 3.367 |
| 3 | 3.401 | 3.434 | 3.466 | 3.497 | 3.526 | 3.555 | 3.584 | 3.611 | 3.638 | 3.664 |
| 4 | 3.689 | 3.714 | 3.738 | 3.761 | 3.784 | 3.807 | 3.829 | 3.850 | 3.871 | 3.892 |
| 5 | 3.912 | 3.932 | 3.951 | 3.970 | 3.989 | 4.007 | 4.025 | 4.043 | 4.060 | 4.078 |
| 6 | 4.094 | 4.111 | 4.127 | 4.143 | 4.159 | 4.174 | 4.190 | 4.205 | 4.220 | 4.234 |
| 7 | 4.249 | 4.263 | 4.277 | 4.291 | 4.304 | 4.318 | 4.331 | 4.344 | 4.357 | 4.369 |
| 8 | 4.382 | 4.394 | 4.407 | 4.419 | 4.431 | 4.443 | 4.454 | 4.466 | 4.477 | 4.489 |
| 9 | 4.500 | 4.511 | 4.522 | 4.533 | 4.543 | 4.554 | 4.564 | 4.575 | 4.585 | 4.595 |
| 10 | 4.605 | 4.615 | 4.625 | 4.635 | 4.644 | 4.654 | 4.663 | 4.673 | 4.682 | 4.691 |
| 11 | 4.701 | 4.710 | 4.719 | 4.727 | 4.736 | 4.745 | 4.754 | 4.762 | 4.771 | 4.779 |
| 12 | 4.788 | 4.796 | 4.804 | 4.812 | 4.820 | 4.828 | 4.836 | 4.844 | 4.852 | 4.860 |
| 13 | 4.868 | 4.875 | 4.883 | 4.890 | 4.898 | 4.905 | 4.913 | 4.920 | 4.927 | 4.935 |
| 14 | 4.942 | 4.949 | 4.956 | 4.963 | 4.970 | 4.977 | 4.984 | 4.990 | 4.997 | 5.004 |
| 15 | 5.011 | 5.017 | 5.024 | 5.030 | 5.037 | 5.043 | 5.050 | 5.056 | 5.063 | 5.069 |
| 16 | 5.075 | 5.081 | 5.088 | 5.094 | 5.100 | 5.106 | 5.112 | 5.118 | 5.124 | 5.130 |
| 17 | 5.136 | 5.142 | 5.148 | 5.153 | 5.159 | 5.165 | 5.171 | 5.176 | 5.182 | 5.187 |
| 18 | 5.193 | 5.199 | 5.204 | 5.210 | 5.215 | 5.220 | 5.226 | 5.231 | 5.236 | 5.242 |
| 19 | 5.247 | 5.252 | 5.258 | 5.263 | 5.268 | 5.273 | 5.278 | 5.283 | 5.288 | 5.293 |
| 20 | 5.298 | 5.303 | 5.308 | 5.313 | 5.318 | 5.323 | 5.328 | 5.333 | 5.338 | 5.342 |

THE EXPONENTIAL e^x For values of x from 0.000 to 0.099

| x | 0 | .001 | .002 | .003 | .004 | .005 | .006 | .007 | .008 | .009 |
|-----|--------|--------|--------|--------|--------|--------|--------|--------|--------|--------|
| .00 | 1.0000 | 1.0010 | 1.0020 | 1.0030 | 1.0040 | 1.0050 | 1.0060 | 1.0070 | 1.0080 | 1.0090 |
| .01 | 1.0101 | 1.0111 | 1.0121 | 1.0131 | 1.0141 | 1.0151 | 1.0161 | 1.0171 | 1.0182 | 1.0191 |
| .02 | 1.0202 | 1.0212 | 1.0222 | 1.0233 | 1.0243 | 1.0253 | 1.0263 | 1.0274 | 1.0284 | 1.0294 |
| .03 | 1.0305 | 1.0315 | 1.0325 | 1.0336 | 1.0346 | 1.0356 | 1.0367 | 1.0377 | 1.0387 | 1.0398 |
| .04 | 1.0408 | 1.0419 | 1.0429 | 1.0439 | 1.0450 | 1.0460 | 1.0471 | 1.0481 | 1.0492 | 1.0502 |
| .05 | 1.0513 | 1.0523 | 1.0534 | 1.0544 | 1.0555 | 1.0565 | 1.0576 | 1.0587 | 1.0597 | 1.0608 |
| .06 | 1.0618 | 1.0629 | 1.0640 | 1.0650 | 1.0661 | 1.0672 | 1.0682 | 1.0693 | 1.0704 | 1.0714 |
| .07 | 1.0725 | 1.0736 | 1.0747 | 1.0757 | 1.0768 | 1.0779 | 1.0790 | 1.0800 | 1.0811 | 1.0822 |
| .08 | 1.0833 | 1.0844 | 1.0855 | 1.0865 | 1.0876 | 1.0887 | 1.0898 | 1.0909 | 1.0920 | 1.0931 |
| .09 | 1.0942 | 1.0953 | 1.0964 | 1.0975 | 1.0986 | 1.0997 | 1.1008 | 1.1019 | 1.1030 | 1.1041 |

For values of x from 0.10 to 2.99

| x | 0 | .01 | .02 | .03 | .04 | .05 | .06 | .07 | .08 | .09 |
|-----|--------|--------|--------|--------|--------|--------|--------|--------|--------|--------|
| 0.1 | 1.1052 | 1.1163 | 1.1275 | 1.1388 | 1.1503 | 1.1618 | 1.1735 | 1.1853 | 1.1972 | 1.2092 |
| 0.2 | 1.2214 | 1.2337 | 1.2461 | 1.2586 | 1.2712 | 1.2840 | 1.2969 | 1.3100 | 1.3231 | 1.3364 |
| 0.3 | 1.3499 | 1.3634 | 1.3771 | 1.3910 | 1.4049 | 1.4191 | 1.4333 | 1.4477 | 1.4623 | 1.4770 |
| 0.4 | 1.4918 | 1.5068 | 1.5220 | 1.5373 | 1.5527 | 1.5683 | 1.5841 | 1.6000 | 1.6161 | 1.6323 |
| 0.5 | 1.6487 | 1.6653 | 1.6820 | 1.6989 | 1.7160 | 1.7333 | 1.7507 | 1.7683 | 1.7860 | 1.8040 |
| 0.6 | 1.8221 | 1.8404 | 1.8589 | 1.8776 | 1.8965 | 1.9155 | 1.9348 | 1.9542 | 1.9739 | 1.9937 |
| 0.7 | 2.0138 | 2.0340 | 2.0544 | 2.0751 | 2.0959 | 2.1170 | 2.1383 | 2.1598 | 2.1815 | 2.2034 |
| 0.8 | 2.2255 | 2.2479 | 2.2705 | 2.2933 | 2.3164 | 2.3396 | 2.3632 | 2.3869 | 2.4109 | 2.4351 |
| 0.9 | 2.4596 | 2.4843 | 2.5093 | 2.5345 | 2.5600 | 2.5857 | 2.6117 | 2.6379 | 2.6645 | 2.6912 |
| 1.0 | 2.7183 | 2.7456 | 2.7732 | 2.8011 | 2.8292 | 2.8577 | 2.8864 | 2.9154 | 2.9447 | 2.9743 |
| 1.1 | 3.0042 | 3.0344 | 3.0649 | 3.0957 | 3.1268 | 3.1582 | 3.1899 | 3.2220 | 3.2544 | 3.2871 |
| 1.2 | 3.3201 | 3.3535 | 3.3872 | 3.4212 | 3.4556 | 3.4903 | 3.5254 | 3.5609 | 3.5966 | 3.6328 |
| 1.3 | 3.6693 | 3.7062 | 3.7434 | 3.7810 | 3.8190 | 3.8574 | 3.8962 | 3.9354 | 3.9749 | 4.0149 |
| 1.4 | 4.0552 | 4.0960 | 4.1371 | 4.1787 | 4.2207 | 4.2631 | 4.3060 | 4.3492 | 4.3929 | 4.4371 |
| 1.5 | 4.4817 | 4.5267 | 4.5722 | 4.6182 | 4.6646 | 4.7115 | 4.7588 | 4.8066 | 4.8550 | 4.9037 |
| 1.6 | 4.9530 | 5.0028 | 5.0531 | 5.1039 | 5.1552 | 5.2070 | 5.2593 | 5.3122 | 5.3656 | 5.4195 |
| 1.7 | 5.4739 | 5.5290 | 5.5845 | 5.6407 | 5.6973 | 5.7546 | 5.8124 | 5.8709 | 5.9299 | 5.9895 |
| 1.8 | 6.0496 | 6.1104 | 6.1719 | 6.2339 | 6.2965 | 6.3598 | 6.4237 | 6.4883 | 6.5535 | 6.6194 |
| 1.9 | 6.6859 | 6.7531 | 6.8210 | 6.8895 | 6.9588 | 7.0287 | 7.0993 | 7.1707 | 7.2427 | 7.3155 |
| 2.0 | 7.3891 | 7.4633 | 7.5383 | 7.6141 | 7.6906 | 7.7679 | 7.8460 | 7.9248 | 8.0045 | 8.0849 |
| 2.1 | 8.1662 | 8.2482 | 8.3311 | 8.4149 | 8.4994 | 8.5849 | 8.6711 | 8.7583 | 8.8463 | 8.9352 |
| 2.2 | 9.0250 | 9.1157 | 9.2073 | 9.2999 | 9.3933 | 9.4877 | 9.5831 | 9.6794 | 9.7767 | 9.8749 |
| 2.3 | 9.9742 | 10.074 | 10.176 | 10.278 | 10.381 | 10.486 | 10.591 | 10.697 | 10.805 | 10.913 |
| 2.4 | 11.023 | 11.134 | 11.246 | 11.359 | 11.473 | 11.588 | 11.705 | 11.822 | 11.941 | 12.061 |
| 2.5 | 12.182 | 12.305 | 12.429 | 12.554 | 12.680 | 12.807 | 12.936 | 13.066 | 13.197 | 13.330 |
| 2.6 | 13.464 | 13.599 | 13.736 | 13.874 | 14.013 | 14.154 | 14.296 | 14.440 | 14.585 | 14.732 |
| 2.7 | 14.880 | 15.029 | 15.180 | 15.333 | 15.487 | 15.643 | 15.800 | 15.959 | 16.119 | 16.281 |
| 2.8 | 16.445 | 16.610 | 16.777 | 16.945 | 17.116 | 17.288 | 17.462 | 17.637 | 17.814 | 17.993 |
| 2.9 | 18.174 | 18.357 | 18.541 | 18.728 | 18.916 | 19.106 | 19.298 | 19.492 | 19.688 | 19.886 |

For values of x from 3.0 to 8.9

| x | 0 | .1 | .2 | .3 | .4 | .5 | .6 | .7 | .8 | .9 |
|-----|--------|--------|--------|--------|--------|--------|--------|--------|--------|--------|
| 3 | 20.086 | 22.198 | 24.533 | 27.113 | 29.964 | 33.115 | 36.598 | 40.447 | 44.701 | 49.402 |
| 4 | 54.598 | 60.340 | 66.686 | 73.700 | 81.451 | 90.017 | 99.484 | 109.95 | 121.51 | 134.29 |
| 5 | 148.41 | 164.02 | 181.27 | 200.34 | 221.41 | 244.69 | 270.43 | 298.87 | 330.30 | 365.04 |
| 6 | 403.43 | 445.86 | 492.75 | 544.57 | 601.85 | 665.14 | 735.10 | 812.41 | 897.85 | 992.27 |
| 7 | 1096.6 | 1212.0 | 1339.4 | 1480.3 | 1636.0 | 1808.0 | 1998.2 | 2208.3 | 2440.6 | 2697.3 |
| 8 | 2981.0 | 3294.5 | 3641.0 | 4023.9 | 4447.1 | 4914.8 | 5431.7 | 6002.9 | 6634.2 | 7332.0 |

LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS

| Degrees | sin | cos | tan | cot | |
|---------|--------|---------|--------|--------|---------|
| 0° 00' | —∞ | 10.0000 | —∞ | +∞ | 90° 00' |
| 0° 10' | 7.4637 | 9.9999 | 7.4637 | 2.5363 | 89° 50' |
| 0° 20' | 7.7648 | 9.9999 | 7.7648 | 2.2352 | 89° 40' |
| 0° 30' | 7.9408 | 9.9999 | 7.9409 | 2.0591 | 89° 30' |
| 0° 40' | 8.0658 | 9.9999 | 8.0658 | 1.9342 | 89° 20' |
| 0° 50' | 8.1627 | 9.9999 | 8.1627 | 1.8373 | 89° 10' |
| 1° 00' | 8.2419 | 9.9999 | 8.2419 | 1.7581 | 89° 00' |
| 1° 10' | 8.3088 | 9.9999 | 8.3089 | 1.6911 | 88° 50' |
| 1° 20' | 8.3668 | 9.9999 | 8.3669 | 1.6331 | 88° 40' |
| 1° 30' | 8.4179 | 9.9999 | 8.4181 | 1.5819 | 88° 30' |
| 1° 40' | 8.4637 | 9.9998 | 8.4638 | 1.5362 | 88° 20' |
| 1° 50' | 8.5050 | 9.9998 | 8.5053 | 1.4947 | 88° 10' |
| 2° 00' | 8.5428 | 9.9997 | 8.5431 | 1.4569 | 88° 00' |
| 2° 10' | 8.5776 | 9.9997 | 8.5779 | 1.4221 | 87° 50' |
| 2° 20' | 8.6097 | 9.9996 | 8.6101 | 1.3899 | 87° 40' |
| 2° 30' | 8.6397 | 9.9996 | 8.6401 | 1.3599 | 87° 30' |
| 2° 40' | 8.6677 | 9.9995 | 8.6682 | 1.3318 | 87° 20' |
| 2° 50' | 8.6940 | 9.9995 | 8.6945 | 1.3055 | 87° 10' |
| 3° 00' | 8.7188 | 9.9994 | 8.7194 | 1.2806 | 87° 00' |
| 3° 10' | 8.7423 | 9.9993 | 8.7429 | 1.2571 | 86° 50' |
| 3° 20' | 8.7645 | 9.9993 | 8.7652 | 1.2348 | 86° 40' |
| 3° 30' | 8.7857 | 9.9992 | 8.7865 | 1.2135 | 86° 30' |
| 3° 40' | 8.8059 | 9.9991 | 8.8067 | 1.1933 | 86° 20' |
| 3° 50' | 8.8251 | 9.9990 | 8.8261 | 1.1739 | 86° 10' |
| 4° 00' | 8.8436 | 9.9989 | 8.8446 | 1.1554 | 86° 00' |
| 4° 10' | 8.8613 | 9.9989 | 8.8624 | 1.1376 | 85° 50' |
| 4° 20' | 8.8783 | 9.9988 | 8.8795 | 1.1205 | 85° 40' |
| 4° 30' | 8.8946 | 9.9987 | 8.8960 | 1.1040 | 85° 30' |
| 4° 40' | 8.9104 | 9.9986 | 8.9118 | 1.0882 | 85° 20' |
| 4° 50' | 8.9256 | 9.9985 | 8.9272 | 1.0728 | 85° 10' |
| 5° 00' | 8.9403 | 9.9983 | 8.9420 | 1.0580 | 85° 00' |
| 5° 10' | 8.9545 | 9.9982 | 8.9563 | 1.0437 | 84° 50' |
| 5° 20' | 8.9682 | 9.9981 | 8.9701 | 1.0299 | 84° 40' |
| 5° 30' | 8.9816 | 9.9980 | 8.9836 | 1.0164 | 84° 30' |
| | cos | sin | cot | tan | Degrees |

LOGARITHMIC SINES, COSINES, TANGENTS,
 AND COTANGENTS (*Continued*)

| Degrees | sin | cos | tan | cot | |
|---------|--------|--------|--------|--------|---------|
| 5° 40' | 8.9945 | 9.9979 | 8.9966 | 1.0034 | 84° 20' |
| 5° 50' | 9.0070 | 9.9977 | 9.0093 | 0.9907 | 84° 10' |
| 6° 00' | 9.0192 | 9.9976 | 9.0216 | 0.9784 | 84° 00' |
| 6° 10' | 9.0311 | 9.9975 | 9.0336 | 0.9664 | 83° 50' |
| 6° 20' | 9.0426 | 9.9973 | 9.0453 | 0.9547 | 83° 40' |
| 6° 30' | 9.0539 | 9.9972 | 9.0567 | 0.9433 | 83° 30' |
| 6° 40' | 9.0648 | 9.9971 | 9.0678 | 0.9322 | 83° 20' |
| 6° 50' | 9.0755 | 9.9969 | 9.0786 | 0.9214 | 83° 10' |
| 7° 00' | 9.0859 | 9.9968 | 9.0891 | 0.9109 | 83° 00' |
| 7° 10' | 9.0961 | 9.9966 | 9.0995 | 0.9005 | 82° 50' |
| 7° 20' | 9.1060 | 9.9964 | 9.1096 | 0.8904 | 82° 40' |
| 7° 30' | 9.1157 | 9.9963 | 9.1194 | 0.8806 | 82° 30' |
| 7° 40' | 9.1252 | 9.9961 | 9.1291 | 0.8709 | 82° 20' |
| 7° 50' | 9.1345 | 9.9959 | 9.1385 | 0.8615 | 82° 10' |
| 8° 00' | 9.1436 | 9.9958 | 9.1478 | 0.8522 | 82° 00' |
| 8° 10' | 9.1525 | 9.9956 | 9.1569 | 0.8431 | 81° 50' |
| 8° 20' | 9.1612 | 9.9954 | 9.1658 | 0.8342 | 81° 40' |
| 8° 30' | 9.1697 | 9.9952 | 9.1745 | 0.8255 | 81° 30' |
| 8° 40' | 9.1781 | 9.9950 | 9.1831 | 0.8169 | 81° 20' |
| 8° 50' | 9.1863 | 9.9948 | 9.1915 | 0.8085 | 81° 10' |
| 9° 00' | 9.1943 | 9.9946 | 9.1997 | 0.8003 | 81° 00' |
| 9° 10' | 9.2022 | 9.9944 | 9.2078 | 0.7922 | 80° 50' |
| 9° 20' | 9.2100 | 9.9942 | 9.2158 | 0.7842 | 80° 40' |
| 9° 30' | 9.2176 | 9.9940 | 9.2236 | 0.7764 | 80° 30' |
| 9° 40' | 9.2251 | 9.9938 | 9.2313 | 0.7687 | 80° 20' |
| 9° 50' | 9.2324 | 9.9936 | 9.2389 | 0.7611 | 80° 10' |
| 10° 00' | 9.2397 | 9.9934 | 9.2463 | 0.7537 | 80° 00' |
| 10° 10' | 9.2468 | 9.9931 | 9.2536 | 0.7464 | 79° 50' |
| 10° 20' | 9.2538 | 9.9929 | 9.2609 | 0.7391 | 79° 40' |
| 10° 30' | 9.2606 | 9.9927 | 9.2680 | 0.7320 | 79° 30' |
| 10° 40' | 9.2674 | 9.9924 | 9.2750 | 0.7250 | 79° 20' |
| 10° 50' | 9.2740 | 9.9922 | 9.2819 | 0.7181 | 79° 10' |
| 11° 00' | 9.2806 | 9.9919 | 9.2887 | 0.7113 | 79° 00' |
| 11° 10' | 9.2870 | 9.9917 | 9.2953 | 0.7047 | 78° 50' |
| | cos | sin | cot | tan | Degrees |

LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS (*Continued*)

| Degrees | sin | cos | tan | cot | |
|---------|--------|--------|--------|--------|---------|
| 11° 20' | 9.2934 | 9.9914 | 9.3020 | 0.6980 | 78° 40' |
| 11° 30' | 9.2997 | 9.9912 | 9.3085 | 0.6915 | 78° 30' |
| 11° 40' | 9.3058 | 9.9909 | 9.3149 | 0.6851 | 78° 20' |
| 11° 50' | 9.3119 | 9.9907 | 9.3212 | 0.6788 | 78° 10' |
| 12° 00' | 9.3179 | 9.9904 | 9.3275 | 0.6725 | 78° 00' |
| 12° 10' | 9.3238 | 9.9901 | 9.3336 | 0.6664 | 77° 50' |
| 12° 20' | 9.3296 | 9.9899 | 9.3397 | 0.6603 | 77° 40' |
| 12° 30' | 9.3353 | 9.9896 | 9.3458 | 0.6542 | 77° 30' |
| 12° 40' | 9.3410 | 9.9893 | 9.3517 | 0.6483 | 77° 20' |
| 12° 50' | 9.3466 | 9.9890 | 9.3576 | 0.6424 | 77° 10' |
| 13° 00' | 9.3521 | 9.9887 | 9.3634 | 0.6366 | 77° 00' |
| 13° 10' | 9.3575 | 9.9884 | 9.3691 | 0.6309 | 76° 50' |
| 13° 20' | 9.3629 | 9.9881 | 9.3748 | 0.6252 | 76° 40' |
| 13° 30' | 9.3682 | 9.9878 | 9.3804 | 0.6196 | 76° 30' |
| 13° 40' | 9.3734 | 9.9875 | 9.3859 | 0.6141 | 76° 20' |
| 13° 50' | 9.3786 | 9.9872 | 9.3914 | 0.6086 | 76° 10' |
| 14° 00' | 9.3837 | 9.9869 | 9.3968 | 0.6032 | 76° 00' |
| 14° 10' | 9.3887 | 9.9866 | 9.4021 | 0.5979 | 75° 50' |
| 14° 20' | 9.3937 | 9.9863 | 9.4074 | 0.5926 | 75° 40' |
| 14° 30' | 9.3986 | 9.9859 | 9.4127 | 0.5873 | 75° 30' |
| 14° 40' | 9.4035 | 9.9856 | 9.4178 | 0.5822 | 75° 20' |
| 14° 50' | 9.4083 | 9.9853 | 9.4230 | 0.5770 | 75° 10' |
| 15° 00' | 9.4130 | 9.9849 | 9.4281 | 0.5719 | 75° 00' |
| 15° 10' | 9.4177 | 9.9846 | 9.4331 | 0.5669 | 74° 50' |
| 15° 20' | 9.4223 | 9.9843 | 9.4381 | 0.5619 | 74° 40' |
| 15° 30' | 9.4269 | 9.9839 | 9.4430 | 0.5570 | 74° 30' |
| 15° 40' | 9.4314 | 9.9836 | 9.4479 | 0.5521 | 74° 20' |
| 15° 50' | 9.4359 | 9.9832 | 9.4527 | 0.5473 | 74° 10' |
| 16° 00' | 9.4403 | 9.9828 | 9.4575 | 0.5425 | 74° 00' |
| 16° 10' | 9.4447 | 9.9825 | 9.4622 | 0.5378 | 73° 50' |
| 16° 20' | 9.4491 | 9.9821 | 9.4669 | 0.5331 | 73° 40' |
| 16° 30' | 9.4533 | 9.9817 | 9.4716 | 0.5284 | 73° 30' |
| 16° 40' | 9.4576 | 9.9814 | 9.4762 | 0.5238 | 73° 20' |
| 16° 50' | 9.4618 | 9.9810 | 9.4808 | 0.5192 | 73° 10' |
| | cos | sin | cot | tan | Degrees |

LOGARITHMIC SINES, COSINES, TANGENTS,
 AND COTANGENTS (*Continued*)

| Degrees | sin | cos | tan | cot | |
|---------|--------|--------|--------|--------|---------|
| 17° 00' | 9.4659 | 9.9806 | 9.4853 | 0.5147 | 73° 00' |
| 17° 10' | 9.4700 | 9.9802 | 9.4898 | 0.5102 | 72° 50' |
| 17° 20' | 9.4741 | 9.9798 | 9.4943 | 0.5057 | 72° 40' |
| 17° 30' | 9.4781 | 9.9794 | 9.4987 | 0.5013 | 72° 30' |
| 17° 40' | 9.4821 | 9.9790 | 9.5031 | 0.4969 | 72° 20' |
| 17° 50' | 9.4861 | 9.9786 | 9.5075 | 0.4925 | 72° 10' |
| 18° 00' | 9.4900 | 9.9782 | 9.5118 | 0.4882 | 72° 00' |
| 18° 10' | 9.4939 | 9.9778 | 9.5161 | 0.4839 | 71° 50' |
| 18° 20' | 9.4977 | 9.9774 | 9.5203 | 0.4797 | 71° 40' |
| 18° 30' | 9.5015 | 9.9770 | 9.5245 | 0.4755 | 71° 30' |
| 18° 40' | 9.5052 | 9.9765 | 9.5287 | 0.4713 | 71° 20' |
| 18° 50' | 9.5090 | 9.9761 | 9.5329 | 0.4671 | 71° 10' |
| 19° 00' | 9.5126 | 9.9757 | 9.5370 | 0.4630 | 71° 00' |
| 19° 10' | 9.5163 | 9.9752 | 9.5411 | 0.4589 | 70° 50' |
| 19° 20' | 9.5199 | 9.9748 | 9.5451 | 0.4549 | 70° 40' |
| 19° 30' | 9.5235 | 9.9743 | 9.5491 | 0.4509 | 70° 30' |
| 19° 40' | 9.5270 | 9.9739 | 9.5531 | 0.4469 | 70° 20' |
| 19° 50' | 9.5306 | 9.9734 | 9.5571 | 0.4429 | 70° 10' |
| 20° 00' | 9.5341 | 9.9730 | 9.5611 | 0.4389 | 70° 00' |
| 20° 10' | 9.5375 | 9.9725 | 9.5650 | 0.4350 | 69° 50' |
| 20° 20' | 9.5409 | 9.9721 | 9.5689 | 0.4311 | 69° 40' |
| 20° 30' | 9.5443 | 9.9716 | 9.5727 | 0.4273 | 69° 30' |
| 20° 40' | 9.5477 | 9.9711 | 9.5766 | 0.4234 | 69° 20' |
| 20° 50' | 9.5510 | 9.9706 | 9.5804 | 0.4196 | 69° 10' |
| 21° 00' | 9.5543 | 9.9702 | 9.5842 | 0.4158 | 69° 00' |
| 21° 10' | 9.5576 | 9.9697 | 9.5879 | 0.4121 | 68° 50' |
| 21° 20' | 9.5609 | 9.9692 | 9.5917 | 0.4083 | 68° 40' |
| 21° 30' | 9.5641 | 9.9687 | 9.5954 | 0.4046 | 68° 30' |
| 21° 40' | 9.5673 | 9.9682 | 9.5991 | 0.4009 | 68° 20' |
| 21° 50' | 9.5704 | 9.9677 | 9.6028 | 0.3972 | 68° 10' |
| 22° 00' | 9.5736 | 9.9672 | 9.6064 | 0.3936 | 68° 00' |
| 22° 10' | 9.5767 | 9.9667 | 9.6100 | 0.3900 | 67° 50' |
| 22° 20' | 9.5798 | 9.9661 | 9.6136 | 0.3864 | 67° 40' |
| 22° 30' | 9.5828 | 9.9656 | 9.6172 | 0.3828 | 67° 30' |
| | cos | sin | cot | tan | Degrees |

LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS (*Continued*)

| Degrees | sin | cos | tan | cot | |
|---------|--------|--------|--------|--------|---------|
| 22° 40' | 9.5859 | 9.9651 | 9.6208 | 0.3792 | 67° 20' |
| 22° 50' | 9.5889 | 9.9646 | 9.6243 | 0.3757 | 67° 10' |
| 23° 00' | 9.5919 | 9.9640 | 9.6279 | 0.3721 | 67° 00' |
| 23° 10' | 9.5948 | 9.9635 | 9.6314 | 0.3686 | 66° 50' |
| 23° 20' | 9.5978 | 9.9629 | 9.6348 | 0.3652 | 66° 40' |
| 23° 30' | 9.6007 | 9.9624 | 9.6383 | 0.3617 | 66° 30' |
| 23° 40' | 9.6036 | 9.9618 | 9.6417 | 0.3583 | 66° 20' |
| 23° 50' | 9.6065 | 9.9613 | 9.6452 | 0.3548 | 66° 10' |
| 24° 00' | 9.6093 | 9.9607 | 9.6486 | 0.3514 | 66° 00' |
| 24° 10' | 9.6121 | 9.9602 | 9.6520 | 0.3480 | 65° 50' |
| 24° 20' | 9.6149 | 9.9596 | 9.6553 | 0.3447 | 65° 40' |
| 24° 30' | 9.6177 | 9.9590 | 9.6587 | 0.3413 | 65° 30' |
| 24° 40' | 9.6205 | 9.9584 | 9.6620 | 0.3380 | 65° 20' |
| 24° 50' | 9.6232 | 9.9579 | 9.6654 | 0.3346 | 65° 10' |
| 25° 00' | 9.6259 | 9.9573 | 9.6687 | 0.3313 | 65° 00' |
| 25° 10' | 9.6286 | 9.9567 | 9.6720 | 0.3280 | 64° 50' |
| 25° 20' | 9.6313 | 9.9561 | 9.6752 | 0.3248 | 64° 40' |
| 25° 30' | 9.6340 | 9.9555 | 9.6785 | 0.3215 | 64° 30' |
| 25° 40' | 9.6366 | 9.9549 | 9.6817 | 0.3183 | 64° 20' |
| 25° 50' | 9.6392 | 9.9543 | 9.6850 | 0.3150 | 64° 10' |
| 26° 00' | 9.6418 | 9.9537 | 9.6882 | 0.3118 | 64° 00' |
| 26° 10' | 9.6444 | 9.9530 | 9.6914 | 0.3086 | 63° 50' |
| 26° 20' | 9.6470 | 9.9524 | 9.6946 | 0.3054 | 63° 40' |
| 26° 30' | 9.6495 | 9.9518 | 9.6977 | 0.3023 | 63° 30' |
| 26° 40' | 9.6521 | 9.9512 | 9.7009 | 0.2991 | 63° 20' |
| 26° 50' | 9.6546 | 9.9505 | 9.7040 | 0.2960 | 63° 10' |
| 27° 00' | 9.6570 | 9.9499 | 9.7072 | 0.2928 | 63° 00' |
| 27° 10' | 9.6595 | 9.9492 | 9.7103 | 0.2897 | 62° 50' |
| 27° 20' | 9.6620 | 9.9486 | 9.7134 | 0.2866 | 62° 40' |
| 27° 30' | 9.6644 | 9.9479 | 9.7165 | 0.2835 | 62° 30' |
| 27° 40' | 9.6668 | 9.9473 | 9.7196 | 0.2804 | 62° 20' |
| 27° 50' | 9.6692 | 9.9466 | 9.7226 | 0.2774 | 62° 10' |
| 28° 00' | 9.6716 | 9.9459 | 9.7257 | 0.2743 | 62° 00' |
| 28° 10' | 9.6740 | 9.9453 | 9.7287 | 0.2713 | 61° 50' |
| | cos | sin | cot | tan | Degrees |

LOGARITHMIC SINES, COSINES, TANGENTS,
 AND COTANGENTS (*Continued*)

| Degrees | sin | cos | tan | cot | |
|---------|--------|--------|--------|--------|---------|
| 28° 20' | 9.6763 | 9.9446 | 9.7317 | 0.2683 | 61° 40' |
| 28° 30' | 9.6787 | 9.9439 | 9.7348 | 0.2652 | 61° 30' |
| 28° 40' | 9.6810 | 9.9432 | 9.7378 | 0.2622 | 61° 20' |
| 28° 50' | 9.6833 | 9.9425 | 9.7408 | 0.2592 | 61° 10' |
| 29° 00' | 9.6856 | 9.9418 | 9.7438 | 0.2562 | 61° 00' |
| 29° 10' | 9.6878 | 9.9411 | 9.7467 | 0.2533 | 60° 50' |
| 29° 20' | 9.6901 | 9.9404 | 9.7497 | 0.2503 | 60° 40' |
| 29° 30' | 9.6923 | 9.9397 | 9.7526 | 0.2474 | 60° 30' |
| 29° 40' | 9.6946 | 9.9390 | 9.7556 | 0.2444 | 60° 20' |
| 29° 50' | 9.6968 | 9.9383 | 9.7585 | 0.2415 | 60° 10' |
| 30° 00' | 9.6990 | 9.9375 | 9.7614 | 0.2386 | 60° 00' |
| 30° 10' | 9.7012 | 9.9368 | 9.7644 | 0.2356 | 59° 50' |
| 30° 20' | 9.7033 | 9.9361 | 9.7673 | 0.2327 | 59° 40' |
| 30° 30' | 9.7055 | 9.9353 | 9.7701 | 0.2299 | 59° 30' |
| 30° 40' | 9.7076 | 9.9346 | 9.7730 | 0.2270 | 59° 20' |
| 30° 50' | 9.7097 | 9.9338 | 9.7759 | 0.2241 | 59° 10' |
| 31° 00' | 9.7118 | 9.9331 | 9.7788 | 0.2212 | 59° 00' |
| 31° 10' | 9.7139 | 9.9323 | 9.7816 | 0.2184 | 58° 50' |
| 31° 20' | 9.7160 | 9.9315 | 9.7845 | 0.2155 | 58° 40' |
| 31° 30' | 9.7181 | 9.9308 | 9.7873 | 0.2127 | 58° 30' |
| 31° 40' | 9.7201 | 9.9300 | 9.7902 | 0.2098 | 58° 20' |
| 31° 50' | 9.7222 | 9.9292 | 9.7930 | 0.2070 | 58° 10' |
| 32° 00' | 9.7242 | 9.9284 | 9.7958 | 0.2042 | 58° 00' |
| 32° 10' | 9.7262 | 9.9276 | 9.7986 | 0.2014 | 57° 50' |
| 32° 20' | 9.7282 | 9.9268 | 9.8014 | 0.1986 | 57° 40' |
| 32° 30' | 9.7302 | 9.9260 | 9.8042 | 0.1958 | 57° 30' |
| 32° 40' | 9.7322 | 9.9252 | 9.8070 | 0.1930 | 57° 20' |
| 32° 50' | 9.7342 | 9.9244 | 9.8097 | 0.1903 | 57° 10' |
| 33° 00' | 9.7361 | 9.9236 | 9.8125 | 0.1875 | 57° 00' |
| 33° 10' | 9.7380 | 9.9228 | 9.8153 | 0.1847 | 56° 50' |
| 33° 20' | 9.7400 | 9.9219 | 9.8180 | 0.1820 | 56° 40' |
| 33° 30' | 9.7419 | 9.9211 | 9.8208 | 0.1792 | 56° 30' |
| 33° 40' | 9.7438 | 9.9203 | 9.8235 | 0.1765 | 56° 20' |
| 33° 50' | 9.7457 | 9.9194 | 9.8263 | 0.1737 | 56° 10' |
| | cos | sin | cot | tan | Degrees |

LOGARITHMIC SINES, COSINES, TANGENTS,
AND COTANGENTS. (Continued)

| Degrees | sin | cos | tan | cot | |
|---------|--------|--------|--------|--------|---------|
| 34° 00' | 9.7476 | 9.9186 | 9.8290 | 0.1710 | 56° 00' |
| 34° 10' | 9.7494 | 9.9177 | 9.8317 | 0.1683 | 55° 50' |
| 34° 20' | 9.7513 | 9.9169 | 9.8344 | 0.1656 | 55° 40' |
| 34° 30' | 9.7531 | 9.9160 | 9.8371 | 0.1629 | 55° 30' |
| 34° 40' | 9.7550 | 9.9151 | 9.8398 | 0.1602 | 55° 20' |
| 34° 50' | 9.7568 | 9.9142 | 9.8425 | 0.1575 | 55° 10' |
| | | | | | |
| 35° 00' | 9.7586 | 9.9134 | 9.8452 | 0.1548 | 55° 00' |
| 35° 10' | 9.7604 | 9.9125 | 9.8479 | 0.1521 | 54° 50' |
| 35° 20' | 9.7622 | 9.9116 | 9.8506 | 0.1494 | 54° 40' |
| 35° 30' | 9.7640 | 9.9107 | 9.8533 | 0.1467 | 54° 30' |
| 35° 40' | 9.7657 | 9.9098 | 9.8559 | 0.1441 | 54° 20' |
| 35° 50' | 9.7675 | 9.9089 | 9.8586 | 0.1414 | 54° 10' |
| | | | | | |
| 36° 00' | 9.7692 | 9.9080 | 9.8613 | 0.1387 | 54° 00' |
| 36° 10' | 9.7710 | 9.9070 | 9.8639 | 0.1361 | 53° 50' |
| 36° 20' | 9.7727 | 9.9061 | 9.8666 | 0.1334 | 53° 40' |
| 36° 30' | 9.7744 | 9.9052 | 9.8692 | 0.1308 | 53° 30' |
| 36° 40' | 9.7761 | 9.9042 | 9.8718 | 0.1282 | 53° 20' |
| 36° 50' | 9.7778 | 9.9033 | 9.8745 | 0.1255 | 53° 10' |
| | | | | | |
| 37° 00' | 9.7795 | 9.9023 | 9.8771 | 0.1229 | 53° 00' |
| 37° 10' | 9.7811 | 9.9014 | 9.8797 | 0.1203 | 52° 50' |
| 37° 20' | 9.7828 | 9.9004 | 9.8824 | 0.1176 | 52° 40' |
| 37° 30' | 9.7844 | 9.8995 | 9.8850 | 0.1150 | 52° 30' |
| 37° 40' | 9.7861 | 9.8985 | 9.8876 | 0.1124 | 52° 20' |
| 37° 50' | 9.7877 | 9.8975 | 9.8902 | 0.1098 | 52° 10' |
| | | | | | |
| 38° 00' | 9.7893 | 9.8965 | 9.8928 | 0.1072 | 52° 00' |
| 38° 10' | 9.7910 | 9.8955 | 9.8954 | 0.1046 | 51° 50' |
| 38° 20' | 9.7926 | 9.8945 | 9.8980 | 0.1020 | 51° 40' |
| 38° 30' | 9.7941 | 9.8935 | 9.9006 | 0.0994 | 51° 30' |
| 38° 40' | 9.7957 | 9.8925 | 9.9032 | 0.0968 | 51° 20' |
| 38° 50' | 9.7973 | 9.8915 | 9.9058 | 0.0942 | 51° 10' |
| | | | | | |
| 39° 00' | 9.7989 | 9.8905 | 9.9084 | 0.0916 | 51° 00' |
| 39° 10' | 9.8004 | 9.8895 | 9.9110 | 0.0890 | 50° 50' |
| 39° 20' | 9.8020 | 9.8884 | 9.9135 | 0.0865 | 50° 40' |
| 39° 30' | 9.8035 | 9.8874 | 9.9161 | 0.0839 | 50° 30' |
| 39° 40' | 9.8050 | 9.8864 | 9.9187 | 0.0813 | 50° 20' |
| 39° 50' | 9.8066 | 9.8853 | 9.9212 | 0.0788 | 50° 10' |
| | cos | sin | cot | tan | Degrees |

LOGARITHMIC SINES, COSINES, TANGENTS,
 AND COTANGENTS (*Continued*)

| Degrees | sin | cos | tan | cot | |
|---------|--------|--------|--------|--------|---------|
| 40° 00' | 9.8081 | 9.8843 | 9.9238 | 0.0762 | 50° 00' |
| 40° 10' | 9.8096 | 9.8832 | 9.9264 | 0.0736 | 49° 50' |
| 40° 20' | 9.8111 | 9.8821 | 9.9289 | 0.0711 | 49° 40' |
| 40° 30' | 9.8125 | 9.8810 | 9.9315 | 0.0685 | 49° 30' |
| 40° 40' | 9.8140 | 9.8800 | 9.9341 | 0.0659 | 49° 20' |
| 40° 50' | 9.8155 | 9.8789 | 9.9366 | 0.0634 | 49° 10' |
| 41° 00' | 9.8169 | 9.8778 | 9.9392 | 0.0608 | 49° 00' |
| 41° 10' | 9.8184 | 9.8767 | 9.9417 | 0.0583 | 48° 50' |
| 41° 20' | 9.8198 | 9.8756 | 9.9443 | 0.0557 | 48° 40' |
| 41° 30' | 9.8213 | 9.8745 | 9.9468 | 0.0532 | 48° 30' |
| 41° 40' | 9.8227 | 9.8733 | 9.9494 | 0.0506 | 48° 20' |
| 41° 50' | 9.8241 | 9.8722 | 9.9519 | 0.0481 | 48° 10' |
| 42° 00' | 9.8255 | 9.8711 | 9.9544 | 0.0456 | 48° 00' |
| 42° 10' | 9.8269 | 9.8699 | 9.9570 | 0.0430 | 47° 50' |
| 42° 20' | 9.8283 | 9.8688 | 9.9595 | 0.0405 | 47° 40' |
| 42° 30' | 9.8297 | 9.8676 | 9.9621 | 0.0379 | 47° 30' |
| 42° 40' | 9.8311 | 9.8665 | 9.9646 | 0.0354 | 47° 20' |
| 42° 50' | 9.8324 | 9.8653 | 9.9671 | 0.0329 | 47° 10' |
| 43° 00' | 9.8338 | 9.8641 | 9.9697 | 0.0303 | 47° 00' |
| 43° 10' | 9.8351 | 9.8629 | 9.9722 | 0.0278 | 46° 50' |
| 43° 20' | 9.8365 | 9.8618 | 9.9747 | 0.0253 | 46° 40' |
| 43° 30' | 9.8378 | 9.8606 | 9.9772 | 0.0228 | 46° 30' |
| 43° 40' | 9.8391 | 9.8594 | 9.9798 | 0.0202 | 46° 20' |
| 43° 50' | 9.8405 | 9.8582 | 9.9823 | 0.0177 | 46° 10' |
| 44° 00' | 9.8418 | 9.8569 | 9.9848 | 0.0152 | 46° 00' |
| 44° 10' | 9.8431 | 9.8557 | 9.9874 | 0.0126 | 45° 50' |
| 44° 20' | 9.8444 | 9.8545 | 9.9899 | 0.0101 | 45° 40' |
| 44° 30' | 9.8457 | 9.8532 | 9.9924 | 0.0076 | 45° 30' |
| 44° 40' | 9.8469 | 9.8520 | 9.9949 | 0.0051 | 45° 20' |
| 44° 50' | 9.8482 | 9.8507 | 9.9975 | 0.0025 | 45° 10' |
| 45° 00' | 9.8495 | 9.8495 | 0.0000 | 0.0000 | 45° 00' |
| | cos | sin | cot | tan | Degrees |

NATURAL SINES, COSINES, TANGENTS, AND COTANGENTS

| Degrees | sin | cos | tan | cot | |
|---------|-------|--------|-------|--------|---------|
| 0° 00' | .0000 | 1.0000 | .0000 | ∞ | 90° 00' |
| 0° 10' | .0029 | 1.0000 | .0029 | 343.77 | 89° 50' |
| 0° 20' | .0058 | 1.0000 | .0058 | 171.89 | 89° 40' |
| 0° 30' | .0087 | 1.0000 | .0087 | 114.59 | 89° 30' |
| 0° 40' | .0116 | .9999 | .0116 | 85.940 | 89° 20' |
| 0° 50' | .0145 | .9999 | .0145 | 68.750 | 89° 10' |
| 1° 00' | .0175 | .9998 | .0175 | 57.290 | 89° 00' |
| 1° 10' | .0204 | .9998 | .0204 | 49.104 | 88° 50' |
| 1° 20' | .0233 | .9997 | .0233 | 42.964 | 88° 40' |
| 1° 30' | .0262 | .9997 | .0262 | 38.188 | 88° 30' |
| 1° 40' | .0291 | .9996 | .0291 | 34.368 | 88° 20' |
| 1° 50' | .0320 | .9995 | .0320 | 31.242 | 88° 10' |
| 2° 00' | .0349 | .9994 | .0349 | 28.636 | 88° 00' |
| 2° 10' | .0378 | .9993 | .0378 | 26.432 | 87° 50' |
| 2° 20' | .0407 | .9992 | .0407 | 24.542 | 87° 40' |
| 2° 30' | .0436 | .9990 | .0437 | 22.904 | 87° 30' |
| 2° 40' | .0465 | .9989 | .0466 | 21.470 | 87° 20' |
| 2° 50' | .0494 | .9988 | .0495 | 20.206 | 87° 10' |
| 3° 00' | .0523 | .9986 | .0524 | 19.081 | 87° 00' |
| 3° 10' | .0552 | .9985 | .0553 | 18.075 | 86° 50' |
| 3° 20' | .0581 | .9983 | .0582 | 17.169 | 86° 40' |
| 3° 30' | .0610 | .9981 | .0612 | 16.350 | 86° 30' |
| 3° 40' | .0640 | .9980 | .0641 | 15.605 | 86° 20' |
| 3° 50' | .0669 | .9978 | .0670 | 14.924 | 86° 10' |
| 4° 00' | .0698 | .9976 | .0699 | 14.301 | 86° 00' |
| 4° 10' | .0727 | .9974 | .0729 | 13.727 | 85° 50' |
| 4° 20' | .0756 | .9971 | .0758 | 13.197 | 85° 40' |
| 4° 30' | .0785 | .9969 | .0787 | 12.706 | 85° 30' |
| 4° 40' | .0814 | .9967 | .0816 | 12.251 | 85° 20' |
| 4° 50' | .0843 | .9964 | .0846 | 11.826 | 85° 10' |
| 5° 00' | .0872 | .9962 | .0875 | 11.430 | 85° 00' |
| 5° 10' | .0901 | .9959 | .0904 | 11.059 | 84° 50' |
| 5° 20' | .0929 | .9957 | .0934 | 10.712 | 84° 40' |
| 5° 30' | .0958 | .9954 | .0963 | 10.385 | 84° 30' |
| | cos | sin | cot | tan | Degrees |

NATURAL SINES, COSINES, TANGENTS, AND
 COTANGENTS (*Continued*)

| Degrees | sin | cos | tan | cot | |
|---------|-------|-------|-------|--------|---------|
| 5° 40' | .0987 | .9951 | .0992 | 10.078 | 84° 20' |
| 5° 50' | .1016 | .9948 | .1022 | 9.7882 | 84° 10' |
| 6° 00' | .1045 | .9945 | .1051 | 9.5144 | 84° 00' |
| 6° 10' | .1074 | .9942 | .1080 | 9.2553 | 83° 50' |
| 6° 20' | .1103 | .9939 | .1110 | 9.0098 | 83° 40' |
| 6° 30' | .1132 | .9936 | .1139 | 8.7769 | 83° 30' |
| 6° 40' | .1161 | .9932 | .1169 | 8.5555 | 83° 20' |
| 6° 50' | .1190 | .9929 | .1198 | 8.3450 | 83° 10' |
| 7° 00' | .1219 | .9925 | .1228 | 8.1443 | 83° 00' |
| 7° 10' | .1248 | .9922 | .1257 | 7.9530 | 82° 50' |
| 7° 20' | .1276 | .9918 | .1287 | 7.7704 | 82° 40' |
| 7° 30' | .1305 | .9914 | .1317 | 7.5958 | 82° 30' |
| 7° 40' | .1334 | .9911 | .1346 | 7.4287 | 82° 20' |
| 7° 50' | .1363 | .9907 | .1376 | 7.2687 | 82° 10' |
| 8° 00' | .1392 | .9903 | .1405 | 7.1154 | 82° 00' |
| 8° 10' | .1421 | .9899 | .1435 | 6.9682 | 81° 50' |
| 8° 20' | .1449 | .9894 | .1465 | 6.8269 | 81° 40' |
| 8° 30' | .1478 | .9890 | .1495 | 6.6912 | 81° 30' |
| 8° 40' | .1507 | .9886 | .1524 | 6.5606 | 81° 20' |
| 8° 50' | .1536 | .9881 | .1554 | 6.4348 | 81° 10' |
| 9° 00' | .1564 | .9877 | .1584 | 6.3138 | 81° 00' |
| 9° 10' | .1593 | .9872 | .1614 | 6.1970 | 80° 50' |
| 9° 20' | .1622 | .9868 | .1644 | 6.0844 | 80° 40' |
| 9° 30' | .1650 | .9863 | .1673 | 5.9758 | 80° 30' |
| 9° 40' | .1679 | .9858 | .1703 | 5.8708 | 80° 20' |
| 9° 50' | .1708 | .9853 | .1733 | 5.7694 | 80° 10' |
| 10° 00' | .1736 | .9848 | .1763 | 5.6713 | 80° 00' |
| 10° 10' | .1765 | .9843 | .1793 | 5.5764 | 79° 50' |
| 10° 20' | .1794 | .9838 | .1823 | 5.4845 | 79° 40' |
| 10° 30' | .1822 | .9833 | .1853 | 5.3955 | 79° 30' |
| 10° 40' | .1851 | .9827 | .1883 | 5.3093 | 79° 20' |
| 10° 50' | .1880 | .9822 | .1914 | 5.2257 | 79° 10' |
| 11° 00' | .1908 | .9816 | .1944 | 5.1446 | 79° 00' |
| 11° 10' | .1937 | .9811 | .1974 | 5.0658 | 78° 50' |
| | cos | sin | cot | tan | Degrees |

**NATURAL SINES, COSINES, TANGENTS, AND
COTANGENTS (Continued)**

| Degrees | sin | cos | tan | cot | |
|---------|-------|-------|-------|--------|---------|
| 11° 20' | .1965 | .9805 | .2004 | 4.9894 | 78° 40' |
| 11° 30' | .1994 | .9799 | .2035 | 4.9152 | 78° 30' |
| 11° 40' | .2022 | .9793 | .2065 | 4.8430 | 78° 20' |
| 11° 50' | .2051 | .9787 | .2095 | 4.7729 | 78° 10' |
| 12° 00' | .2079 | .9781 | .2126 | 4.7046 | 78° 00' |
| 12° 10' | .2108 | .9775 | .2156 | 4.6382 | 77° 50' |
| 12° 20' | .2136 | .9769 | .2186 | 4.5736 | 77° 40' |
| 12° 30' | .2164 | .9763 | .2217 | 4.5107 | 77° 30' |
| 12° 40' | .2193 | .9757 | .2247 | 4.4494 | 77° 20' |
| 12° 50' | .2221 | .9750 | .2278 | 4.3897 | 77° 10' |
| 13° 00' | .2250 | .9744 | .2309 | 4.3315 | 77° 00' |
| 13° 10' | .2278 | .9737 | .2339 | 4.2747 | 76° 50' |
| 13° 20' | .2306 | .9730 | .2370 | 4.2193 | 76° 40' |
| 13° 30' | .2334 | .9724 | .2401 | 4.1653 | 76° 30' |
| 13° 40' | .2363 | .9717 | .2432 | 4.1126 | 76° 20' |
| 13° 50' | .2391 | .9710 | .2462 | 4.0611 | 76° 10' |
| 14° 00' | .2419 | .9703 | .2493 | 4.0108 | 76° 00' |
| 14° 10' | .2447 | .9696 | .2524 | 3.9617 | 75° 50' |
| 14° 20' | .2476 | .9689 | .2555 | 3.9136 | 75° 40' |
| 14° 30' | .2504 | .9681 | .2586 | 3.8667 | 75° 30' |
| 14° 40' | .2532 | .9674 | .2617 | 3.8208 | 75° 20' |
| 14° 50' | .2560 | .9667 | .2648 | 3.7760 | 75° 10' |
| 15° 00' | .2588 | .9659 | .2679 | 3.7321 | 75° 00' |
| 15° 10' | .2616 | .9652 | .2711 | 3.6891 | 74° 50' |
| 15° 20' | .2644 | .9644 | .2742 | 3.6470 | 74° 40' |
| 15° 30' | .2672 | .9636 | .2773 | 3.6059 | 74° 30' |
| 15° 40' | .2700 | .9628 | .2805 | 3.5656 | 74° 20' |
| 15° 50' | .2728 | .9621 | .2836 | 3.5261 | 74° 10' |
| 16° 00' | .2756 | .9613 | .2867 | 3.4874 | 74° 00' |
| 16° 10' | .2784 | .9605 | .2899 | 3.4495 | 73° 50' |
| 16° 20' | .2812 | .9596 | .2931 | 3.4124 | 73° 40' |
| 16° 30' | .2840 | .9588 | .2962 | 3.3759 | 73° 30' |
| 16° 40' | .2868 | .9580 | .2994 | 3.3402 | 73° 20' |
| 16° 50' | .2896 | .9572 | .3026 | 3.3052 | 73° 10' |
| | cos | sin | cot | tan | Degrees |

NATURAL SINES, COSINES, TANGENTS, AND
COTANGENTS (*Continued*)

| Degrees | sin | cos | tan | cot | |
|---------|-------|-------|-------|--------|---------|
| 17° 00' | .2924 | .9563 | .3057 | 3.2709 | 73° 00' |
| 17° 10' | .2952 | .9555 | .3089 | 3.2371 | 72° 50' |
| 17° 20' | .2979 | .9546 | .3121 | 3.2041 | 72° 40' |
| 17° 30' | .3007 | .9537 | .3153 | 3.1716 | 72° 30' |
| 17° 40' | .3035 | .9528 | .3185 | 3.1397 | 72° 20' |
| 17° 50' | .3062 | .9520 | .3217 | 3.1084 | 72° 10' |
| 18° 00' | .3090 | .9511 | .3249 | 3.0777 | 72° 00' |
| 18° 10' | .3118 | .9502 | .3281 | 3.0475 | 71° 50' |
| 18° 20' | .3145 | .9492 | .3314 | 3.0178 | 71° 40' |
| 18° 30' | .3173 | .9483 | .3346 | 2.9887 | 71° 30' |
| 18° 40' | .3201 | .9474 | .3378 | 2.9600 | 71° 20' |
| 18° 50' | .3228 | .9465 | .3411 | 2.9319 | 71° 10' |
| 19° 00' | .3256 | .9455 | .3443 | 2.9042 | 71° 00' |
| 19° 10' | .3283 | .9446 | .3476 | 2.8770 | 70° 50' |
| 19° 20' | .3311 | .9436 | .3508 | 2.8502 | 70° 40' |
| 19° 30' | .3338 | .9426 | .3541 | 2.8239 | 70° 30' |
| 19° 40' | .3365 | .9417 | .3574 | 2.7980 | 70° 20' |
| 19° 50' | .3393 | .9407 | .3607 | 2.7725 | 70° 10' |
| 20° 00' | .3420 | .9397 | .3640 | 2.7475 | 70° 00' |
| 20° 10' | .3448 | .9387 | .3673 | 2.7228 | 69° 50' |
| 20° 20' | .3475 | .9377 | .3706 | 2.6985 | 69° 40' |
| 20° 30' | .3502 | .9367 | .3739 | 2.6746 | 69° 30' |
| 20° 40' | .3529 | .9356 | .3772 | 2.6511 | 69° 20' |
| 20° 50' | .3557 | .9346 | .3805 | 2.6279 | 69° 10' |
| 21° 00' | .3584 | .9336 | .3839 | 2.6051 | 69° 00' |
| 21° 10' | .3611 | .9325 | .3872 | 2.5826 | 68° 50' |
| 21° 20' | .3638 | .9315 | .3906 | 2.5605 | 68° 40' |
| 21° 30' | .3665 | .9304 | .3939 | 2.5386 | 68° 30' |
| 21° 40' | .3692 | .9293 | .3973 | 2.5172 | 68° 20' |
| 21° 50' | .3719 | .9283 | .4006 | 2.4960 | 68° 10' |
| 22° 00' | .3746 | .9272 | .4040 | 2.4751 | 68° 00' |
| 22° 10' | .3773 | .9261 | .4074 | 2.4545 | 67° 50' |
| 22° 20' | .3800 | .9250 | .4108 | 2.4342 | 67° 40' |
| 22° 30' | .3827 | .9239 | .4142 | 2.4142 | 67° 30' |
| | cos | sin | cot | tan | Degrees |

NATURAL SINES, COSINES, TANGENTS, AND
COTANGENTS (*Continued*)

| Degrees | sin | cos | tan | cot | |
|---------|-------|-------|-------|--------|---------|
| 22° 40' | .3854 | .9228 | .4176 | 2.3945 | 67° 20' |
| 22° 50' | .3881 | .9216 | .4210 | 2.3750 | 67° 10' |
| 23° 00' | .3907 | .9205 | .4245 | 2.3559 | 67° 00' |
| 23° 10' | .3934 | .9194 | .4279 | 2.3369 | 66° 50' |
| 23° 20' | .3961 | .9182 | .4314 | 2.3183 | 66° 40' |
| 23° 30' | .3987 | .9171 | .4348 | 2.2998 | 66° 30' |
| 23° 40' | .4014 | .9159 | .4383 | 2.2817 | 66° 20' |
| 23° 50' | .4041 | .9147 | .4417 | 2.2637 | 66° 10' |
| 24° 00' | .4067 | .9135 | .4452 | 2.2460 | 66° 00' |
| 24° 10' | .4094 | .9124 | .4487 | 2.2286 | 65° 50' |
| 24° 20' | .4120 | .9112 | .4522 | 2.2113 | 65° 40' |
| 24° 30' | .4147 | .9100 | .4557 | 2.1943 | 65° 30' |
| 24° 40' | .4173 | .9088 | .4592 | 2.1775 | 65° 20' |
| 24° 50' | .4200 | .9075 | .4628 | 2.1609 | 65° 10' |
| 25° 00' | .4226 | .9063 | .4663 | 2.1445 | 65° 00' |
| 25° 10' | .4253 | .9051 | .4699 | 2.1283 | 64° 50' |
| 25° 20' | .4279 | .9038 | .4734 | 2.1123 | 64° 40' |
| 25° 30' | .4305 | .9026 | .4770 | 2.0965 | 64° 30' |
| 25° 40' | .4331 | .9013 | .4806 | 2.0809 | 64° 20' |
| 25° 50' | .4358 | .9001 | .4841 | 2.0655 | 64° 10' |
| 26° 00' | .4384 | .8988 | .4877 | 2.0503 | 64° 00' |
| 26° 10' | .4410 | .8975 | .4913 | 2.0353 | 63° 50' |
| 26° 20' | .4436 | .8962 | .4950 | 2.0204 | 63° 40' |
| 26° 30' | .4462 | .8949 | .4986 | 2.0057 | 63° 30' |
| 26° 40' | .4488 | .8936 | .5022 | 1.9912 | 63° 20' |
| 26° 50' | .4514 | .8923 | .5059 | 1.9768 | 63° 10' |
| 27° 00' | .4540 | .8910 | .5095 | 1.9626 | 63° 00' |
| 27° 10' | .4566 | .8897 | .5132 | 1.9486 | 62° 50' |
| 27° 20' | .4592 | .8884 | .5169 | 1.9347 | 62° 40' |
| 27° 30' | .4617 | .8870 | .5206 | 1.9210 | 62° 30' |
| 27° 40' | .4643 | .8857 | .5243 | 1.9074 | 62° 20' |
| 27° 50' | .4669 | .8843 | .5280 | 1.8940 | 62° 10' |
| 28° 00' | .4695 | .8829 | .5317 | 1.8807 | 62° 00' |
| 28° 10' | .4720 | .8816 | .5354 | 1.8676 | 61° 50' |
| | cos | sin | cot | tan | Degrees |

NATURAL SINES, COSINES, TANGENTS, AND
COTANGENTS (*Continued*)

| Degrees | sin | cos | tan | cot | |
|---------|-------|-------|-------|--------|---------|
| 28° 20' | .4746 | .8802 | .5392 | 1.8546 | 61° 40' |
| 28° 30' | .4772 | .8788 | .5430 | 1.8418 | 61° 30' |
| 28° 40' | .4797 | .8774 | .5467 | 1.8291 | 61° 20' |
| 28° 50' | .4823 | .8760 | .5505 | 1.8165 | 61° 10' |
| 29° 00' | .4848 | .8746 | .5543 | 1.8040 | 61° 00' |
| 29° 10' | .4874 | .8732 | .5581 | 1.7917 | 60° 50' |
| 29° 20' | .4899 | .8718 | .5619 | 1.7796 | 60° 40' |
| 29° 30' | .4924 | .8704 | .5658 | 1.7675 | 60° 30' |
| 29° 40' | .4950 | .8689 | .5696 | 1.7556 | 60° 20' |
| 29° 50' | .4975 | .8675 | .5735 | 1.7437 | 60° 10' |
| 30° 00' | .5000 | .8660 | .5774 | 1.7321 | 60° 00' |
| 30° 10' | .5025 | .8646 | .5812 | 1.7205 | 59° 50' |
| 30° 20' | .5050 | .8631 | .5851 | 1.7090 | 59° 40' |
| 30° 30' | .5075 | .8616 | .5890 | 1.6977 | 59° 30' |
| 30° 40' | .5100 | .8601 | .5930 | 1.6864 | 59° 20' |
| 30° 50' | .5125 | .8587 | .5969 | 1.6753 | 59° 10' |
| 31° 00' | .5150 | .8572 | .6009 | 1.6643 | 59° 00' |
| 31° 10' | .5175 | .8557 | .6048 | 1.6534 | 58° 50' |
| 31° 20' | .5200 | .8542 | .6088 | 1.6426 | 58° 40' |
| 31° 30' | .5225 | .8526 | .6128 | 1.6319 | 58° 30' |
| 31° 40' | .5250 | .8511 | .6168 | 1.6212 | 58° 20' |
| 31° 50' | .5275 | .8496 | .6208 | 1.6107 | 58° 10' |
| 32° 00' | .5299 | .8480 | .6249 | 1.6003 | 58° 00' |
| 32° 10' | .5324 | .8465 | .6289 | 1.5900 | 57° 50' |
| 32° 20' | .5348 | .8450 | .6330 | 1.5798 | 57° 40' |
| 32° 30' | .5373 | .8434 | .6371 | 1.5697 | 57° 30' |
| 32° 40' | .5398 | .8418 | .6412 | 1.5597 | 57° 20' |
| 32° 50' | .5422 | .8403 | .6453 | 1.5497 | 57° 10' |
| 33° 00' | .5446 | .8387 | .6494 | 1.5399 | 57° 00' |
| 33° 10' | .5471 | .8371 | .6536 | 1.5301 | 56° 50' |
| 33° 20' | .5495 | .8355 | .6577 | 1.5204 | 56° 40' |
| 33° 30' | .5519 | .8339 | .6619 | 1.5108 | 56° 30' |
| 33° 40' | .5544 | .8323 | .6661 | 1.5013 | 56° 20' |
| 33° 50' | .5568 | .8307 | .6703 | 1.4919 | 56° 10' |
| | cos | sin | cot | tan | Degrees |

NATURAL SINES, COSINES, TANGENTS, AND
COTANGENTS (*Continued*)

| Degrees | sin | cos | tan | cot | |
|---------|-------|-------|-------|--------|---------|
| 34° 00' | .5592 | .8290 | .6745 | 1.4826 | 56° 00' |
| 34° 10' | .5616 | .8274 | .6787 | 1.4733 | 55° 50' |
| 34° 20' | .5640 | .8258 | .6830 | 1.4641 | 55° 40' |
| 34° 30' | .5664 | .8241 | .6873 | 1.4550 | 55° 30' |
| 34° 40' | .5688 | .8225 | .6916 | 1.4460 | 55° 20' |
| 34° 50' | .5712 | .8208 | .6959 | 1.4370 | 55° 10' |
| 35° 00' | .5736 | .8192 | .7002 | 1.4281 | 55° 00' |
| 35° 10' | .5760 | .8175 | .7046 | 1.4193 | 54° 50' |
| 35° 20' | .5783 | .8158 | .7089 | 1.4106 | 54° 40' |
| 35° 30' | .5807 | .8141 | .7133 | 1.4019 | 54° 30' |
| 35° 40' | .5831 | .8124 | .7177 | 1.3934 | 54° 20' |
| 35° 50' | .5854 | .8107 | .7221 | 1.3848 | 54° 10' |
| 36° 00' | .5878 | .8090 | .7265 | 1.3764 | 54° 00' |
| 36° 10' | .5901 | .8073 | .7310 | 1.3680 | 53° 50' |
| 36° 20' | .5925 | .8056 | .7355 | 1.3597 | 53° 40' |
| 36° 30' | .5948 | .8039 | .7400 | 1.3514 | 53° 30' |
| 36° 40' | .5972 | .8021 | .7445 | 1.3432 | 53° 20' |
| 36° 50' | .5995 | .8004 | .7490 | 1.3351 | 53° 10' |
| 37° 00' | .6018 | .7986 | .7536 | 1.3270 | 53° 00' |
| 37° 10' | .6041 | .7969 | .7581 | 1.3190 | 52° 50' |
| 37° 20' | .6065 | .7951 | .7627 | 1.3111 | 52° 40' |
| 37° 30' | .6088 | .7934 | .7673 | 1.3032 | 52° 30' |
| 37° 40' | .6111 | .7916 | .7720 | 1.2954 | 52° 20' |
| 37° 50' | .6134 | .7898 | .7766 | 1.2876 | 52° 10' |
| 38° 00' | .6157 | .7880 | .7813 | 1.2799 | 52° 00' |
| 38° 10' | .6180 | .7862 | .7860 | 1.2723 | 51° 50' |
| 38° 20' | .6202 | .7844 | .7907 | 1.2647 | 51° 40' |
| 38° 30' | .6225 | .7826 | .7954 | 1.2572 | 51° 30' |
| 38° 40' | .6248 | .7808 | .8002 | 1.2497 | 51° 20' |
| 38° 50' | .6271 | .7790 | .8050 | 1.2423 | 51° 10' |
| 39° 00' | .6293 | .7771 | .8098 | 1.2349 | 51° 00' |
| 39° 10' | .6316 | .7753 | .8146 | 1.2276 | 50° 50' |
| 39° 20' | .6338 | .7735 | .8195 | 1.2203 | 50° 40' |
| 39° 30' | .6361 | .7716 | .8243 | 1.2131 | 50° 30' |
| | cos | sin | cot | tan | Degrees |

NATURAL SINES, COSINES, TANGENTS, AND
COTANGENTS (*Continued*)

| Degrees | sin | cos | tan | cot | |
|---------|-------|-------|--------|--------|---------|
| 39° 40' | .6383 | .7698 | .8292 | 1.2059 | 50° 20' |
| 39° 50' | .6406 | .7679 | .8342 | 1.1988 | 50° 10' |
| 40° 00' | .6428 | .7660 | .8391 | 1.1918 | 50° 00' |
| 40° 10' | .6450 | .7642 | .8441 | 1.1847 | 49° 50' |
| 40° 20' | .6472 | .7623 | .8491 | 1.1778 | 49° 40' |
| 40° 30' | .6494 | .7604 | .8541 | 1.1708 | 49° 30' |
| 40° 40' | .6517 | .7585 | .8591 | 1.1640 | 49° 20' |
| 40° 50' | .6539 | .7566 | .8642 | 1.1571 | 49° 10' |
| 41° 00' | .6561 | .7547 | .8693 | 1.1504 | 49° 00' |
| 41° 10' | .6583 | .7528 | .8744 | 1.1436 | 48° 50' |
| 41° 20' | .6604 | .7509 | .8796 | 1.1369 | 48° 40' |
| 41° 30' | .6626 | .7490 | .8847 | 1.1303 | 48° 30' |
| 41° 40' | .6648 | .7470 | .8899 | 1.1237 | 48° 20' |
| 41° 50' | .6670 | .7451 | .8952 | 1.1171 | 48° 10' |
| 42° 00' | .6691 | .7431 | .9004 | 1.1106 | 48° 00' |
| 42° 10' | .6713 | .7412 | .9057 | 1.1041 | 47° 50' |
| 42° 20' | .6734 | .7392 | .9110 | 1.0977 | 47° 40' |
| 42° 30' | .6756 | .7373 | .9163 | 1.0913 | 47° 30' |
| 42° 40' | .6777 | .7353 | .9217 | 1.0850 | 47° 20' |
| 42° 50' | .6799 | .7333 | .9271 | 1.0786 | 47° 10' |
| 43° 00' | .6820 | .7314 | .9325 | 1.0724 | 47° 00' |
| 43° 10' | .6841 | .7294 | .9380 | 1.0661 | 46° 50' |
| 43° 20' | .6862 | .7274 | .9435 | 1.0599 | 46° 40' |
| 43° 30' | .6884 | .7254 | .9490 | 1.0538 | 46° 30' |
| 43° 40' | .6905 | .7234 | .9545 | 1.0477 | 46° 20' |
| 43° 50' | .6926 | .7214 | .9601 | 1.0416 | 46° 10' |
| 44° 00' | .6947 | .7193 | .9657 | 1.0355 | 46° 00' |
| 44° 10' | .6967 | .7173 | .9713 | 1.0295 | 45° 50' |
| 44° 20' | .6988 | .7153 | .9770 | 1.0235 | 45° 40' |
| 44° 30' | .7009 | .7133 | .9827 | 1.0176 | 45° 30' |
| 44° 40' | .7030 | .7112 | .9884 | 1.0117 | 45° 20' |
| 44° 50' | .7050 | .7092 | .9942 | 1.0058 | 45° 10' |
| 45° 00' | .7071 | .7071 | 1.0000 | 1.0000 | 45° 00' |
| | cos | sin | cot | tan | Degrees |

HYPERBOLIC SINES AND COSINES

| x | $\cosh x$ | $\sinh x$ | x | $\cosh x$ | $\sinh x$ |
|------|-----------|-----------|------|-----------|-----------|
| 0.00 | 1.0000 | 0.0000 | 2.05 | 3.9484 | 3.8196 |
| 0.05 | 1.0013 | 0.0500 | 2.10 | 4.1443 | 4.0219 |
| 0.10 | 1.0050 | 0.1002 | 2.15 | 4.3507 | 4.2342 |
| 0.15 | 1.0112 | 0.1506 | 2.20 | 4.5679 | 4.4571 |
| 0.20 | 1.0201 | 0.2013 | 2.25 | 4.7966 | 4.6912 |
| 0.25 | 1.0314 | 0.2526 | 2.30 | 5.0372 | 4.9369 |
| 0.30 | 1.0453 | 0.3045 | 2.35 | 5.2905 | 5.1952 |
| 0.35 | 1.0619 | 0.3572 | 2.40 | 5.5569 | 5.4662 |
| 0.40 | 1.0811 | 0.4108 | 2.45 | 5.8373 | 5.7510 |
| 0.45 | 1.1030 | 0.4653 | 2.50 | 6.1323 | 6.0502 |
| 0.50 | 1.1276 | 0.5211 | 2.55 | 6.4426 | 6.3645 |
| 0.55 | 1.1551 | 0.5782 | 2.60 | 6.7690 | 6.6947 |
| 0.60 | 1.1855 | 0.6367 | 2.65 | 7.1123 | 7.0417 |
| 0.65 | 1.2188 | 0.6967 | 2.70 | 7.4735 | 7.4063 |
| 0.70 | 1.2552 | 0.7586 | 2.75 | 7.8533 | 7.7894 |
| 0.75 | 1.2947 | 0.8223 | 2.80 | 8.2527 | 8.1919 |
| 0.80 | 1.3374 | 0.8881 | 2.85 | 8.6728 | 8.6150 |
| 0.85 | 1.3835 | 0.9561 | 2.90 | 9.1146 | 9.0596 |
| 0.90 | 1.4331 | 1.0265 | 2.95 | 9.5791 | 9.5268 |
| 0.95 | 1.4862 | 1.0995 | 3.00 | 10.0677 | 10.0179 |
| 1.00 | 1.5431 | 1.1752 | 3.05 | 10.5814 | 10.5340 |
| 1.05 | 1.6038 | 1.2539 | 3.10 | 11.1215 | 11.0765 |
| 1.10 | 1.6685 | 1.3356 | 3.15 | 11.6895 | 11.6466 |
| 1.15 | 1.7374 | 1.4208 | 3.20 | 12.2866 | 12.2459 |
| 1.20 | 1.8107 | 1.5097 | 3.25 | 12.9146 | 12.8758 |
| 1.25 | 1.8884 | 1.6019 | 3.30 | 13.5748 | 13.5379 |
| 1.30 | 1.9709 | 1.6984 | 3.35 | 14.2689 | 14.2338 |
| 1.35 | 2.0583 | 1.7991 | 3.40 | 14.9987 | 14.9654 |
| 1.40 | 2.1509 | 1.9043 | 3.45 | 15.7661 | 15.7343 |
| 1.45 | 2.2488 | 2.0143 | 3.50 | 16.5728 | 16.5426 |
| 1.50 | 2.3524 | 2.1293 | 3.55 | 17.4210 | 17.3923 |
| 1.55 | 2.4619 | 2.2496 | 3.60 | 18.3128 | 18.2855 |
| 1.60 | 2.5775 | 2.3757 | 3.65 | 19.2503 | 19.2243 |
| 1.65 | 2.6995 | 2.5075 | 3.70 | 20.2360 | 20.2113 |
| 1.70 | 2.8283 | 2.6456 | 3.75 | 21.2723 | 21.2488 |
| 1.75 | 2.9642 | 2.7904 | 3.80 | 22.3618 | 22.3394 |
| 1.80 | 3.1075 | 2.9422 | 3.85 | 23.5072 | 23.4859 |
| 1.85 | 3.2583 | 3.1013 | 3.90 | 24.7113 | 24.6911 |
| 1.90 | 3.4177 | 3.2682 | 3.95 | 25.9773 | 25.9581 |
| 1.95 | 3.5855 | 3.4432 | 4.00 | 27.3082 | 27.2899 |
| 2.00 | 3.7622 | 3.6269 | | | |

HYPERBOLIC TANGENTS AND COTANGENTS

| x | $\tanh x$ | $\coth x$ | x | $\tanh x$ | $\coth x$ |
|------|-----------|-----------|-------|-----------|-----------|
| 0.00 | 0.00000 | ∞ | | | |
| 0.05 | 0.04996 | 20.017 | 2.05 | 0.96740 | 1.0337 |
| 0.10 | 0.09967 | 10.033 | 2.10 | 0.97045 | 1.0304 |
| 0.15 | 0.14889 | 6.7166 | 2.15 | 0.97323 | 1.0275 |
| 0.20 | 0.19738 | 5.0665 | 2.20 | 0.97574 | 1.0249 |
| 0.25 | 0.24492 | 4.0830 | 2.25 | 0.97803 | 1.0225 |
| 0.30 | 0.29131 | 3.4327 | 2.30 | 0.98010 | 1.0203 |
| 0.35 | 0.33638 | 2.9729 | 2.35 | 0.98197 | 1.0184 |
| 0.40 | 0.37995 | 2.6319 | 2.40 | 0.98367 | 1.0166 |
| 0.45 | 0.42190 | 2.3702 | 2.45 | 0.98522 | 1.0150 |
| 0.50 | 0.46212 | 2.1640 | 2.50 | 0.98661 | 1.0136 |
| 0.55 | 0.50052 | 1.9979 | 2.55 | 0.98788 | 1.0123 |
| 0.60 | 0.53705 | 1.8620 | 2.60 | 0.98903 | 1.0111 |
| 0.65 | 0.57167 | 1.7493 | 2.65 | 0.99007 | 1.0100 |
| 0.70 | 0.60437 | 1.6546 | 2.70 | 0.99101 | 1.0091 |
| 0.75 | 0.63515 | 1.5744 | 2.75 | 0.99186 | 1.0082 |
| 0.80 | 0.66404 | 1.5059 | 2.80 | 0.99263 | 1.0074 |
| 0.85 | 0.69107 | 1.4470 | 2.85 | 0.99333 | 1.0067 |
| 0.90 | 0.71630 | 1.3961 | 2.90 | 0.99396 | 1.0061 |
| 0.95 | 0.73978 | 1.3517 | 2.95 | 0.99454 | 1.0055 |
| 1.00 | 0.76159 | 1.3130 | 3.00 | 0.99505 | 1.0050 |
| 1.05 | 0.78181 | 1.2791 | 3.0 | 0.99505 | 1.0050 |
| 1.10 | 0.80050 | 1.2492 | 3.1 | 0.99595 | 1.0041 |
| 1.15 | 0.81775 | 1.2229 | 3.2 | 0.99668 | 1.0033 |
| 1.20 | 0.83365 | 1.1995 | 3.3 | 0.99728 | 1.0027 |
| 1.25 | 0.84828 | 1.1789 | 3.4 | 0.99777 | 1.0022 |
| 1.30 | 0.86172 | 1.1605 | 3.5 | 0.99818 | 1.0018 |
| 1.35 | 0.87405 | 1.1441 | 3.6 | 0.99851 | 1.0015 |
| 1.40 | 0.88535 | 1.1295 | 3.7 | 0.99878 | 1.0012 |
| 1.45 | 0.89569 | 1.1165 | 3.8 | 0.99900 | 1.0010 |
| 1.50 | 0.90515 | 1.1048 | 3.9 | 0.99918 | 1.0008 |
| 1.55 | 0.91379 | 1.0943 | 4.0 | 0.99933 | 1.0007 |
| 1.60 | 0.92167 | 1.0850 | 4.1 | 0.99945 | 1.0005 |
| 1.65 | 0.92886 | 1.0766 | 4.2 | 0.99955 | 1.0004 |
| 1.70 | 0.93541 | 1.0691 | 4.3 | 0.99963 | 1.0004 |
| 1.75 | 0.94138 | 1.0623 | 4.4 | 0.99970 | 1.0003 |
| 1.80 | 0.94681 | 1.0562 | 4.5 | 0.99975 | 1.0002 |
| 1.85 | 0.95175 | 1.0507 | 4.6 | 0.99980 | 1.0002 |
| 1.90 | 0.95624 | 1.0458 | 4.7 | 0.99983 | 1.0002 |
| 1.95 | 0.96032 | 1.0413 | 4.8 | 0.99986 | 1.0001 |
| 2.00 | 0.96403 | 1.0373 | 4.9 | 0.99989 | 1.0001 |

Numerical Constants

$$\pi = 3.141\ 592\ 654$$

$$\log_{10} \pi = 0.497\ 149\ 873$$

$$\frac{1}{\pi} = 0.318\ 309\ 886$$

$$\pi^2 = 9.869\ 604\ 401$$

$$\sqrt{\pi} = 1.772\ 453\ 851$$

$$\frac{1}{\sqrt{\pi}} = 0.564\ 189\ 583$$

$$\sqrt{\frac{\pi}{2}} = 1.253\ 314\ 137$$

$$\sqrt{\frac{2}{\pi}} = 0.797\ 884\ 561$$

$$e = 2.718\ 281\ 828$$

$$\frac{1}{e} = 0.367\ 879\ 441$$

$$\log_{10} e = 0.434\ 294\ 482$$

$$\log_e 10 = 2.302\ 585\ 093$$

$$\log_{10} \log_{10} e = 9.637\ 784\ 311$$

$$\log_e \pi = 1.144\ 729\ 886$$

$$\log_e 2 = 0.693\ 147\ 181$$

$$\log_{10} 2 = 0.301\ 029\ 996$$

INDEX

A

- Acceleration of gravity, 102
Acre, 196
Aeronautics, 180-192
Air, flow in pipes, 154-155
—, compressed, 155
Airplanes, 182-192
Algebra, 1
Alternating currents, 169-176
Altitudes, determination by barometer, 202
Altitude, value of g , 97
Analytic geometry, 24-39
Angle, between two lines, 25
Angular measure, 98-99
Apertures, flow of water, 142-143
Apothecaries' fluid measure, 194
Apothecaries' weight, 194, 197, 198
Archimedes' spiral, 36
Arc, 196
Area, bounded by curves, 48-49
—, of circles, 13, 226-227
—, of plane figures, 12-14
Arithmetical progressions, 3
Asymptotes of hyperbola, 30
Atmosphere, pressure of, 201
Atomic weights, 204
Avoirdupois weight, 194

B

- Balloons, 180-182
Barometer, corrections for, 201-202
Base, of logarithms, 4, 5
Bazin's formula, 149-150

- Beams, flexure of, 112-114
Beam loadings, 115-123
Bending moment, 113
Binomial theorem, 2
Boiling-points, 210-211
British thermal unit, 200

C

- Calculus, 40-71
Cantilever beam, 116
Capacity, electric, 167-169, 172
—, measures of, 194
Cardioid, 32
Catenary, 33, 74
Center of gravity, 88-91, 134-136
—, of arc of curve, 88-89
—, of plane figures 89-90, 91, 134-136
—, of solids, 90, 136
Center of gyration, 93-94
Center of percussion, 94
Centigrade scale, 199
Centimeter, 195
Centrifugal force, 99-100
Channels, 128
Characteristic of a logarithm, 231-232
Charge, of condensers, 167
Chemical constants, 204-225
Circle, circumference and area of, 13, 226-227
—, equation of, 26
—, quadrant of, 134
—, sector of, 13
—, segment of, 13
—, tangent to, 27

- Circular measure, 194
 —, mil, 195
 —, motion, 99
 Circumference of circles, 13, 226-227
 Cissoid, 34
 Clairaut's equation, 79
 Climbing, of airplanes, 189-192
 Coefficients, of friction, 105, 106
 —, of linear expansion, 215-217
 Collapsing of tubes, 153
 Columns, 129-133
 —, wooden, 133
 Combinations, 11, 12
 Common logarithms, 4, 231, 232
 — —, tables, 233-252
 Comparison test, 6, 7,
 Complementary function, 80
 Complex imaginary quantities, 9-11
 Compressed air, 155
 Compression, 109
 Concentrated loads, 117, 118, 119, 120
 Condensers, electric, 167-169
 Cone, 15
 —, center of gravity of, 136
 Conjugate hyperbola, 30
 Constants, numerical, 276
 Convergent series, 6
 Coordinates, 26
 Copper wire tables, 158-159
 Cosecant, 15, 16
 Cosine, 15, 16
 Cotangent, 15, 16
 Cubes of numbers, 228-230
 Cubical parabola, 35
 Cubic equations, 5, 6, 51
 Curvature, 41
 —, radius of, 41
 Curves, length of, 46-48
 Curve tracing, 52-53
 Cycloid, 30-31
 Cylinder, right circular, 14
 — pressure in, 152
- D**
- De Moivre's theorem, 11
 Deflection of a beam, 114
 Densities of substances, 205-209
 Differential calculus, 40-46
 —, equations, 75-88
 —, formulæ, 42-43
 Direction cosines, 37, 38
 Directrix of parabola, 27
 Discriminant, 1
 Distance, between two points, 25, 38
 —, from point to straight line, 25
 Distortion, 111
 Divergent series, 6
 Dry measure, 194
 Dyne, 198
- E**
- e, base of natural logarithms, 4, 5, 7
 e^x , 256
 e^{-x} , 257
 Earth, diameter of, 203
 Eddy current loss, 178
 Effective voltage, 164, 170
 Elastic curve, 113
 —, limit, 108, 110
 Electric equivalents, 199-201
 Electricity, 156-176
 Ellipse, area of, 14
 —, circumference of, 14
 —, equation of, 29
 — quadrant of, 134
 Energy, 102, 103
 —, electric, 161
 —, kinetic, 103
 English weights and measures, 193-194
 Epicycloid, 31
 Equations, cubic, 5, 6, 51
 —, differential, 75-88
 —, quadratic, 1
 —, solution of, 51
 —, transcendental, 6
 Erg, 199
 Euler's formula for columns, 129
 Exact differential equations, 77, 86
 Exponential curve, 36
 —, functions, 256-257
 —, series, 7
 Exponents, 1

F

- Factor of safety, 108, 110
 Farad, 167
 Fahrenheit scale, 199
 Falling bodies, 96-97
 Fixed beam, 122, 123
 Flexure of beams, 112-114
 Fliegner's equation, 153
 Flow of fluids, 153-155
 Flow of water, in open channels,
 149-151
 —, in pipes, 143-149
 —, over weirs, 151-152
 —, through apertures, 142-143
 Fluid measure, 194
 Fluids, flow of, 153-155
 Flywheel, 100
 Focus, of ellipse, 29
 —, of hyperbola, 29
 —, of parabola, 27
 Force, 102, 103
 —, centrifugal, 99-100
 Forces, composition of, 104
 Friction, 105
 —, coefficients of, 105
 —, of belts, 106
 Frustum, of cone, 15
 —, of pyramid, 14

G

- g, acceleration of gravity, 96, 97,
 198
 Gallon, 197
 Gas, flow in pipes, 154
 Generators, electric, 163-164
 Geometrical progression, 3
 Geometry, 12-15
 Gordon's formula for columns, 130
 Grain, 197
 Gram, 197
 Graphs, 52-53
 Gravity, 96, 97, 198
 —, center of, 134-136
 Gyration, center of, 93, 100
 —, radius of, 93-94, 100, 138-140

H

- Head of water, 141
 Heat equivalents, 199-201

- Hectare, 196
 Hemisphere, 136
 Homogeneous differential equa-
 tions, 75-76, 85-86
 Hydraulics, 141-155
 Hyperbola, 29
 Hyperbolic formulæ, 71-73
 —, functions, 71-75, 274-275
 —, logarithms, 4, 253-255
 Hypocycloid, 32
 Hysteresis loss, 177

I

- I-beams, 124-125
 Imaginary numbers, 9
 Impact, 103, 104
 Impedance, electric, 171, 172
 Inclined plane, 106-108
 Indeterminate forms, 2
 —, evaluation of, 50
 Induced voltage, 164, 165
 Inductance, 165-167, 172
 Inertia, moment of, 91-93, 137-
 140
 Infinite series, 6-8
 Integrals, table of, 57-71
 Integrating factors, 78
 Integration, methods, of, 53-57
 Intercept equation of line, 25
 Inverse hyperbolic functions, 73
 Involute of circle, 33

J

- j, imaginary unit, 9, 10
 Johnson's formula for columns,
 131-132
 Joule, 199

K

- Kutter's formula, 150-151

L

- Latitude, value of g, 97
 Lemniscate, 34
 Length, measures of, 193
 —, of curves, 46-48
 Linear differential equations, 76,
 77, 80-85
 — expansion of solids, 215-217

- Liquid measure, 194
 Loadings of beams, 115-123
 Logarithmic cross-section paper,
 230-231
 —, curve, 36
 —, trigonometric functions, 258-
 265
 Logarithms, 4
 —, common, 233-252
 —, natural, 253-255
 —, use of tables, 231-232
 Loss of pressure in pipes, 144-147

M

- Maclaurin's series, 45
 Magnetism, 176-179
 Mantissa of a logarithm, 231-232
 Maxima and minima, 43-45
 Measurement, tables, 193-201
 Mechanical equivalents, 199-201
 Mechanics of materials, 108-140
 Melting points of elements, 210-
 211
 Meter, 195
 Metric conversion tables, 195-201
 Microfarad, 167
 Millimeter, 195
 Minimum, 43-45
 Modulus of elasticity, 108, 110
 —, in shear, 111
 Mollier steam chart, after 218
 Moment of inertia, 91-93
 —, of plane figures, 91-93
 —, of solids, 93, 137
 Moment of resistance, 113
 Momentum, 102
 Moon, distance from earth, 203
 Motion of a body, 94-95
 —, circular, 99
 Motors, electric, 161-162

N

- Napierian logarithms, 4, 253-255
 Napier's equations, 154
 Natural hyperbolic functions, 274-
 275
 —, logarithms, 4, 253-255

- Natural trigonometric functions,
 266-273
 Navigation, 22
 Neutral axis, 112
 —, surface, 112
 Newton's method of solving equa-
 tions, 51
 Normal lines, 40, 41
 Numerical constants, 276

O

- Oblique triangles, 19-21
 —, spherical, 21-22
 Ohm, 156, 160
 Ohm's law, 160
 Oscillating series, 7
 Oscillation of pendulum, 101
 —, radius of, 94

P

- Parabola, area of, 13
 —, cubical, 35
 —, equation of, 27, 28
 —, length of arc, 13
 —, semi-cubical, 35
 Parallel circuits, electric, 160
 Parallelogram, area of, 12
 —, center of gravity, 134
 Partial fractions, 53
 Particular integral, 80
 Parts, integration by, 54-55
 Pendulum, 101, 102
 Percussion, center of, 94
 Permanent set, 108
 Permeability, magnetic, 176
 Permutations, 11
 Physical constants, 204-225
 Pipes, pressure in, 152
 Plane, analytic geometry, 24-36
 —, equation of, 39
 —, geometry, 12-14
 —, inclined, 106-108
 —, triangles, solution of, 19-21
 —, trigonometry, 15-21
 Polar coordinates, 26
 Polygon, 13
 Pound, 198
 Poundal, 198

Power, electric, 161, 174, 175
— factor, 174

Powers of numbers, 228-230

Pressure, center of, 141-142

—, corrections, 201-202

—, equivalents, 201

—, water, 141

Prism, 14

Progressions, arithmetical, 3

—, geometrical, 3

Projectiles, 97-98

Proportion, 2

Pyramid, 14

—, center of gravity of, 136

—, frustum of, 14

Q

Quadrant, 17

—, of circle, 134

—, of ellipse, 134

Quadratic equations, 1

R

Radian, 98-99

Radius of curvature, 41

—, of gyration, 93-94, 100, 138-140

—, of oscillation, 94

Rankine's formula for columns, 130

Ratio test, 7

Reactance, electric, 171, 172

Reciprocals of numbers, 228-230

Rectangular coordinates, 26

Reduction formulæ for integration, 56-57

Regular polygon, 13

Reluctance, magnetic, 176

Resistance, electric, 156, 157, 160, 161

—, moment of, 113

Resistivity, electric, 156

Restitution, coefficient of, 103

Resultant of forces, 105

Right cone, 15

— cylinder, 14

— prism, 14

— triangle, 12, 15-16, 21

Ritter's formula for columns, 131

Roots of equation 1

—, of numbers, 228-230

Rotation, 96

S

Safety, factor of, 108, 110

Saturated steam tables, 218-225

Saturation curves, magnetic, 178-179

Secant, 15, 16

Seconds-pendulum, 101, 102

Section modulus, 138-140

Sector, of circle, 13, 135

—, of circular ring, 135

Segment, of circle, 13, 135

—, of parabola, 136

—, of sphere, 15

Semi-circle, 135

Semi-cubical parabola, 35

Series circuits, electric, 161

Series, exponential, 7

—, infinite, 6-8

—, logarithmic, 8

Shear, 111

—, horizontal 115

—, vertical, 114-115

Signs, of trigonometric functions, 17

Simple pendulum, 101, 102

Sine, 15, 16

—, waves, 172-174

Slope equation of straight line, 25

—, of normal, 40

—, of tangent, 40

Solid analytic geometry, 37-39

—, geometry, 14-15

Solids of revolution, 49-50

Solution of triangles, 19-21

Sound, velocity of, 203

Specific heats of elements, 212-213

—, of gases, 214

—, of liquids, 213

—, of miscellaneous substances, 215

Sphere, segment of, 15

—, surface area of, 15

—, volume of, 15

Spherical triangles, 21-22

—, trigonometry, 21-24

Spiral of Archimedes, 36
 Steam, chart, after 218
 —, flow in pipes, 155
 —, tables, 218-225
 Steel, strength of, 110
 Stewart's equations, 153
 Straight line, 39
 —, equation of, 24, 25
 —, formula for columns, 132-133
 Strength of materials, 110
 Stress, 108, 109, 114
 Strophoid, 35
 Subnormal, 41, 42
 Substitution, integration by, 55-56
 Subtangent, 41, 42
 Sun, distance from earth, 203
 Surface, measures of, 193
 Surfaces of revolution, 49

T

Tangent, 15, 16, 40, 41
 —, to circle, 27
 —, to ellipse, 29
 —, to hyperbola, 30
 —, to parabola, 28
 Taylor's series, 45
 Temperature coefficient, electric, 157
 — scales, 199
 Tension, 109
 Test, for convergency, 6, 7,
 Theoretical mechanics, 88-108
 Three-phase circuits, 175-176
 Timber, strength of, 110, 133
 Ton, 198
 Tortion, 111
 — of circular shafts, 111-112
 Tracing curves, 52-53
 Transcendental equations, 6
 Transformation of coordinates, 26
 Trapezoid, area of, 12
 —, center of gravity, 134

Triangle, area of, 12, 25
 —, center of gravity, 134
 —, solution of, 19-21
 Trigonometric formulæ, 17-19
 —, logarithmic functions, 258-265
 —, natural functions, 266-273
 Trigonometry, plane, 15-21
 —, spherical, 21-24
 Troy weight, 194, 197, 198
 Tubes, collapsing of, 153

U

Ultimate strength, 108, 110
 Uniform acceleration, 95
 —, load's, 117, 121, 123
 Units, 193, 194

V

Velocity of light, 203
 —, of sound, 203
 Voltage, 164, 165
 Volt-ampere, 174
 Volume corrections, 201-202
 —, measures of, 193

W

Watt, 200
 Weights and measures, 193-194
 Weights of elements, 205-206
 —, of gases, 208
 —, of liquids, 209
 —, of miscellaneous substances, 207
 —, of woods, 208
 Weirs, 151-152
 Wing characteristics of air-planes, 182
 Wire, copper, tables, 158-159
 Witch of Agnes, 34
 Work, 102, 103

Z

Zone, of sphere, 134

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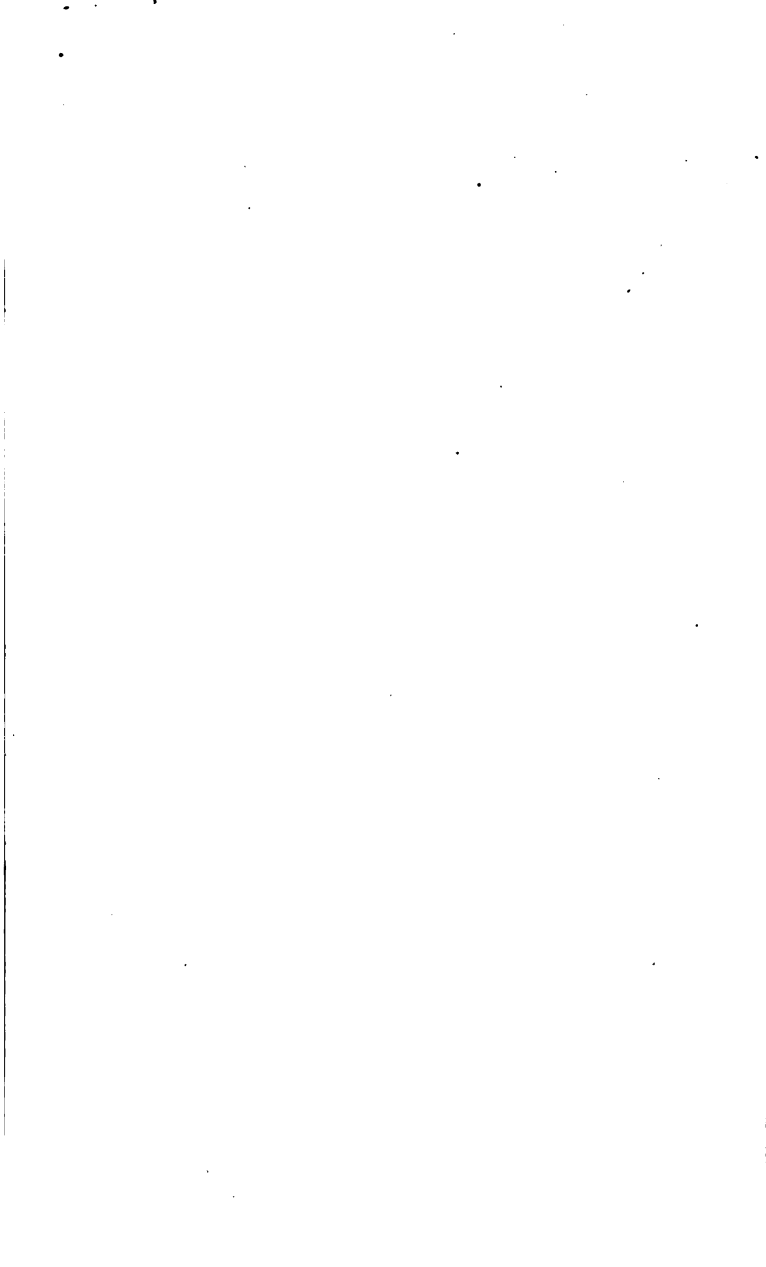
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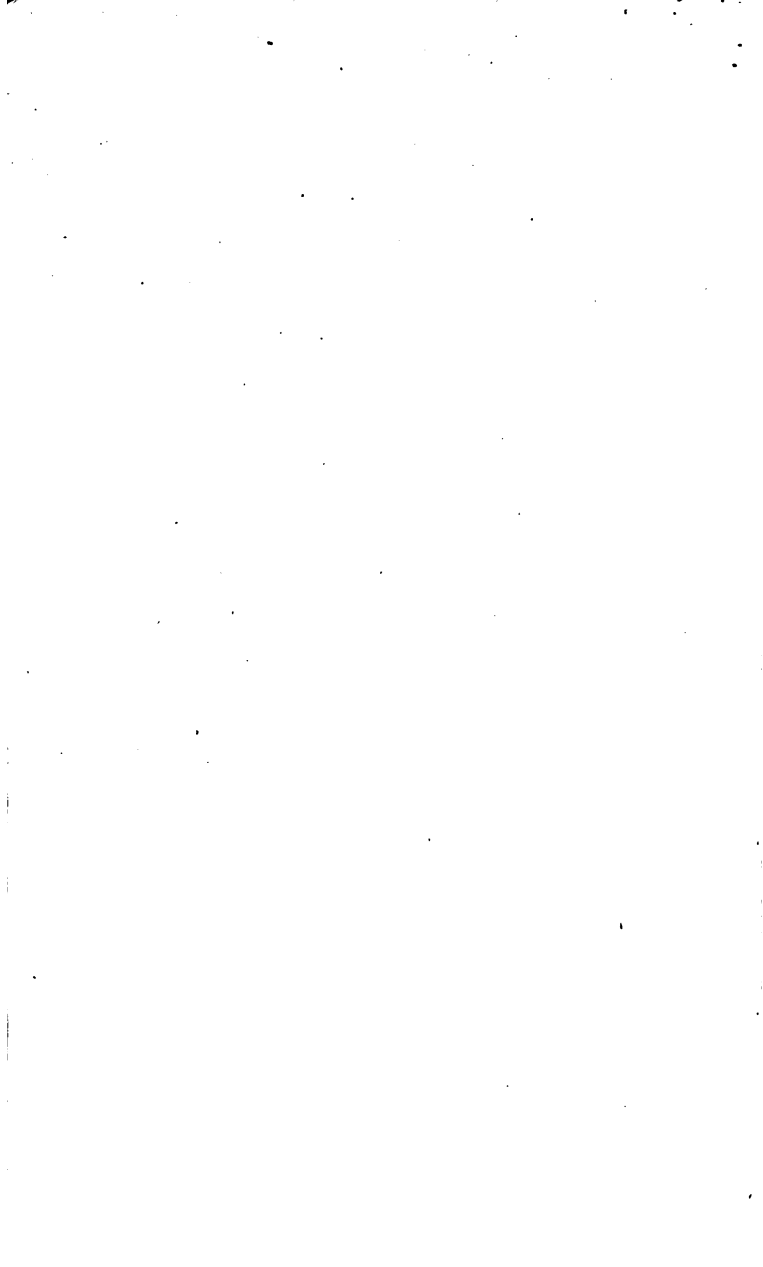
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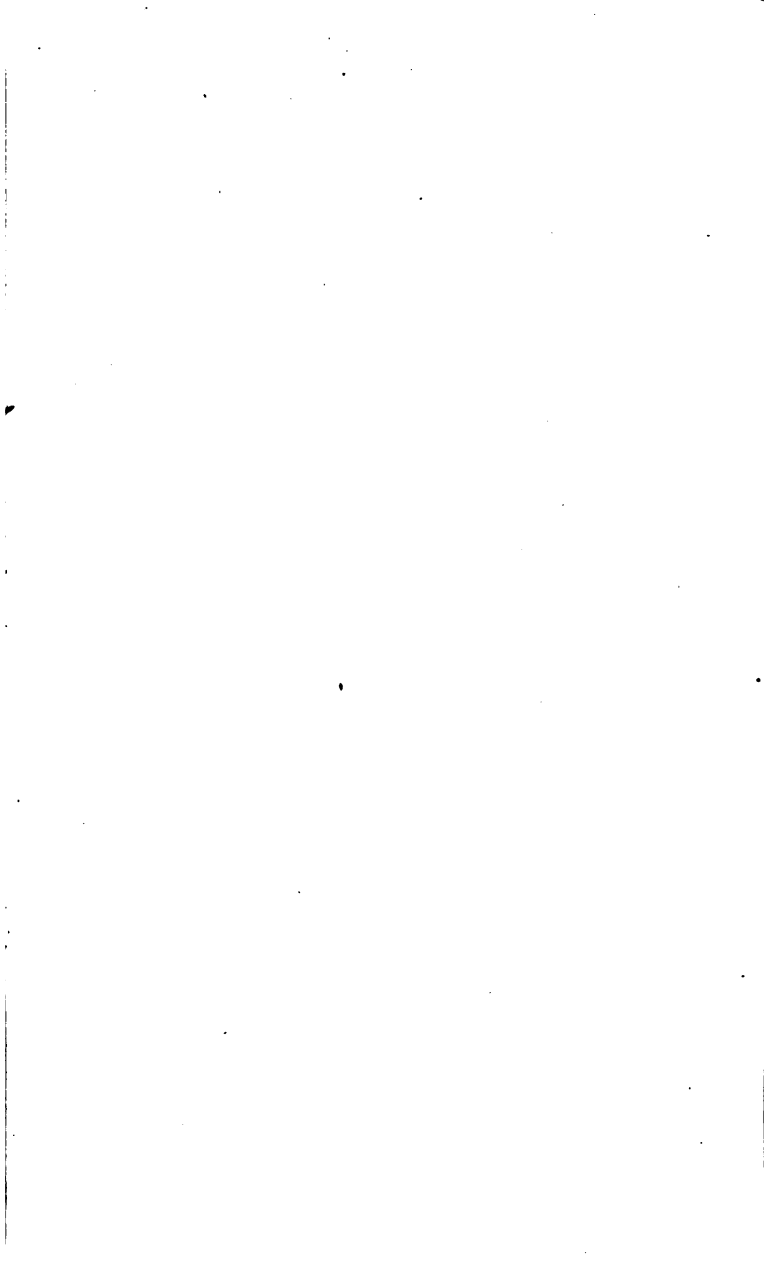
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